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Paracha, Obed-ur-Rahman

AN EXPERIMENTAL INVESTIGATION OF THE EFFECT OF CARBON DIOXIDE ON STEAM DRIVE RECOVERY

The University of Oklahoma

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THE UNIVERSITY OF OKLAHOMA

GRADUATE COLLEGE

AN EXPERIMENTAL INVESTIGATION

OF THE EFFECT OF CARBON DIOXIDE ON STEAM DRIVE

RECOVERY

A DISSERTATION

SUBMITTED TO THE GRADUATE FACULTY

in partial fulfillment of the requirements for the

degree of

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APPROVED FOR THE DEPARTMENT OF PETROLEUM AND GEOLOGICAL ENGINEERING

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ABSTRACT

A laboratory study was undertaken to evaluate the effects of carbon dioxide injection with steam on heavy to intermediate oil recovery. The effects of other operating parameters, such as pressure, temperature, injection rate, oil viscosity, and pH on oil recovery were also investigated.

A large number of displacement tests were conducted on unconsolidated sand packs of 3 inches in diameter and 24 inches in length. The sand packs were saturated with crude oils of gravities 15, 20, and 26°API. A mixture of varying concentration of carbon dioxide in steam, ranging from 0.0 to 0.006 SCF $CO_2/cm.^3$ steam (water equivalent), was injected at different steam temperatures, rates, and pH to evaluate the effect of each individual parameter on oil recovery.

The examination of the results obtained indicated that: (1) the injection of carbon dioxide with steam increases the rate of recovery significantly, (2) the recovery is affected by the concentration of carbon dioxide in the injected steam and is maximized at a concentration of about 0.004 standard cubic feet of carbon dioxide per cubic centimeter of cold water equivalent steam, (3) the overall recovery depends on oil viscosity and hence the API gravity. It improves by 8 % in case of 15°API oil, 4% in case of 20%API oil, whereas no

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significant improvement in ultimate recovery, over the conventional steam flooding process, was observed in case of 26°API oil, (4) the recovery decreases with increasing pressure and hence the temperature, (5) the recovery is rate dependent and is maximized at a steam injection rate of 30 cm.³/minute, (6) the recovery is not affected by pH, when steam and carbon dioxide are injected simultaneously.

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CHAPTER I

INTRODUCTION

Throughout the entire world a considerable amount of oil is classified as unrecoverable by existing primary and secondary recovery techniques, The magnitude of this resource together with the declining world oil reserves prompted the petroleum engineers and researchers to develop enhanced oil recovery methods, which are separated into three major categories:

- 1. Thermal recovery
 - a. Steam stimulation
 - b. Steam drive
 - c. In situ combustion
- 2. Chemical flooding
 - Surfactant/Polymer Injection а.
 - b. Polymer Floodingc. Caustic Flooding
- 3. Miscible Displacement
 - a. Miscible Hydrocarbon Displacement
 - b. CO₂ Injection

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c. Inert Gas Injection

Steam drive and CO_2 injection are the most widespread commercial processes. Steam flooding, the most profitable enhanced oil recovery method today, has proved a powerful recovery process in heavy oil reservoirs. Recently it was applied successfully to a relatively low viscosity oil reservoir.^{\perp}

Carbon dioxide flooding is the next most promising among the enhanced oil recovery methods. Miscible displacement with CO2 has been applied successfully to reservoir oils with API gravities of greater than 25° API, while its application with low API gravity oils has been restricted because of the complex phase behavior, including the possible deposition of asphaltene, which might damage the permeability of the reservoir. Immiscible displacement with CO2 applies efficiently to the heavier, viscous, reservoir oils through the mechanisms of oil swelling and oil viscosity reduction. For all heavy oil reservoirs the major mechanism of the enhanced oil recovery method is the reduction of reservoir oil viscosity which results in a pronounced increase in the mobility of the oils and a corresponding improvement of oil production rates. Welker and Dunlop² published the viscosity reduction effects of CO_2 in 1963. They concluded that viscosity reduction is as high as 98% for a 4800-cp heavy crude oil at 80°F.

Pursley³ and Weinstein⁴ were the first to suggest the addition of gas to cyclic steam stimulation. They found a 50%, or more, increase in oil recovery from gas/steam stimulation compared with steam alone in physical and computer models. They also matched field results for steam stimulation with and without gas. Recently Redford⁵ investigated the effects of solvent addition to steam with highly positive results.

The motivation for this study comes from the promising results of earlier studies mentioned above and the enhanced oil recovery projects where exhaust gases from the steam generators are injected into the reservoir along with the steam. The

emphasis of this study will be on the recovery by steam/CO $_2$ flooding processes.

Statement of the Problem

1. Steam Flooding:

The two most widely used and profitable enhanced oil recovery techniques available today are cyclic steam stimulation and steam flooding (or steam drive). Current oil production from these methods exceeds 550,000 barrels per day which accounts for more than 80% of the total enhanced oil recovery production. Considering the huge proven reserves of heavy viscous oil, discovered to date, which exceed one trillion barrels, the potential for future production by these methods is still higher.

Cyclic steam stimulation process consists of injecting steam into the producing well for a certain specified period of time followed by shutting-in the well to allow sufficient time for the heat to dissipate and spread into the reservoir and then placing the well on production. This process allows the immediate surroundings of the production well to be maintained at a higher temperature thus improving the flow of oil near the well bore. Oil production stabilizes at a much higher level due to the pronounced increase in the mobility of the heavy oil. In addition to the viscosity reduction resulting from steam injection, other factors contributing to the stimulated production are:

a. Thermal expansion

b. Compression of solution gas

c. Well bore clean-up effects

In a steam flooding process, steam is injected into a number of injection wells and oil is produced from the adjacent wells. As the steam moves forward towards the producing well, its temperature drops and at some distance from the injection well it starts condensing forming a hot water bank. The hot water condensed from the steam tends to settle below the steam vapor because of its relative higher density and the steam travels preferentially along the top of the bed as it moves towards the producing well.

Three principal zones develop in this process which are identified as:

- 1. Saturated steam zone
- 2. Condensation zone
- 3. Hot water zone

Each of these zones makes positive contributions towards the enhancement of oil displacement. In the saturated steam region, oil displacement is enhanced by effects of steam distillation, gas drive and solvent extraction, in addition to the viscosity reduction, thermal expansion and reduction in residual oil saturation etc. active in other heated regions too. Oil displacement is also enhanced by the increasing relative permeability to oil with increased temperature.^{6,7,8}

There is little or no information in published literature on failures of the steam flooding process, which would have given valuable data in defining the limits of applications of these processes. Among the parameters to be considered before any practical application, the following are of particular

importance.

- Permeability should be high, no lower than 1 darcy for full scale displacement.
- 2. Oil in place should be about 1200-1700 bbl/acre-foot.
- 3. The oil gravity should be in the range of 15-30°API.
- 4. Formation thickness should be greater than 30 ft. and its depth should be less than 3000 ft. to minimize heat losses. The reservoir depth is also limited by the technical aspects of high pressure injection.
- 2. Carbon Dioxide Flooding:

The idea of oil displacement by CO_2 originated during the late twenties.^{9,10} Through intensive laboratory research and field tests this idea turned into a proven recovery process with the potential of recovering more than 90% of the oil contacted in a reservoir. The factors contributing to the enhanced production as a result of CO_2 injection are identified as:

- 1. Oil Swelling
- 2. Viscosity Reduction
- 3. Miscibility Effects
- 4. Solution Gas Drive
- 5. Reaction with Reservoir Rock

The various mechanisms by which it displaces oil from the porous media include:

- 1. Miscible Drive
- 2. Immiscible Drive
- 3. Trapped Gas Effect

A miscible displacement is one in which the displacing and the displaced fluids become miscible in all proportions, at least to a local extent, without formation of an interface between the two fluids. Miscibility depends upon the pressure, temperature, the composition of the oil and the composition of the displacing fluid. Miscible displacement theoretically recovers all of the reservoir oil contacted because of the elimination of the capillary and interfacial forces which are responsible for retaining substantial quantities of oil under immiscible conditions.

Carbon dioxide has the potential of not only eliminating the capillary and interfacial forces above certain reservoir pressures and thus creating miscible flow conditions, it also is, highly soluble in crude oils at moderate pressures which causes considerable swelling and reduction in the viscosity, thereby increasing recovery efficiency (see figures, 1.1, & 1.2).^{12,13}

The pressure range in this type of displacements varies from about 700 psia necessary to achieve substantial CO_2 solubility, to a certain higher value (ranging from 2000 to more than 5000 psi depending on composition of oil and reservoir temperature) at which the solubility of CO_2 causes sufficient extraction of hydrocarbons to promote miscible displacement. This higher pressure is termed by some investigators¹⁴ as the minimum miscibility pressure and is defined as the pressure at which the recovery is 94% of the oil contacted at a given temperature and above which essentially no additional oil is recovered. A number of correlations are available in the literature for determining

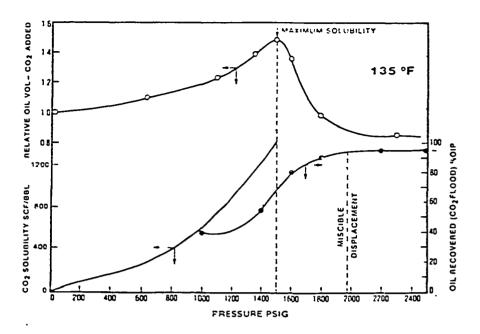


Figure 1.1: Mead Strawn Crude oil (after Ref.12)

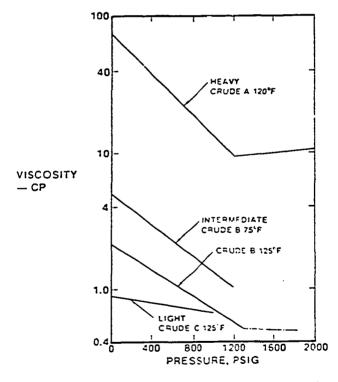


Figure 1.2: Viscosity of Crude oils Saturated with CO₂ (after Ref.13)

the optimum pressure required for maximum oil displacement efficiency by miscible displacement. 15,16,17

It has been reported that lower miscibility pressure is required for lighter oils, while for heavier oils the pressure requirement is quite high to create miscibility conditions (see fig. 1.3)¹⁸ The purity of the injected CO_2 , also affects the miscibility pressure. Its contamination with N_2 or CH_4 causes an increase, whereas C_3H_8 or H_2 causes a decrease in the pressure required for miscible displacement. Reservoir oils with gravities of more than 25° API are the best condidates for CO_2 miscible displacement.

Since the pressure and amount of CO_2 required for miscible displacement of heavier oils is too high, the economic factors do not dictate this type of displacement process. These oils can still be recovered by immiscible CO_2 displacement through the mechanisms of oil swelling and oil viscosity reduction because of the high solubility of CO_2 at reservoir pressures. Oils of gravities as low as 15° API are efficiently recovered by CO_2 immiscible displacement.

During the above mentioned displacement processes by CO_2 (miscible or immiscible), some oil is also recovered by the trapped gas effect. The injection of CO_2 creates a free gas saturation which replaces a part of the residual oil that would not have been otherwise recovered.

Applications of the CO₂ displacement processes also have some limitations given as follows:

1. The principal difficulty in any CO₂ displacement

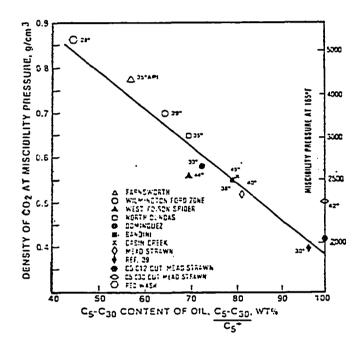


Figure 1.3: Carbon dioxide density required for miscible displacement vs. C_5 through C_{30} content of crude oils (after Ref. 17).

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process is contact of a large fraction of the oil. Because of the very high mobility¹⁹ of CO₂ due to its very low viscosity, the displacement of oil is unstable giving rise to the following problems:

- CO_2 fingers through the more viscous fluid Sweep efficiency is lower than desirable Early breakthrough of CO_2 into producing wells
- Ъ.

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Consequently substantial amount of reservoir oil is not contacted, not swelled and its viscosity is not reduced.

- 2. Thin pay zones and low vertical permeability are preferred to prevent gravity override.
- 3. Oil saturations should be greater than 25% and its gravity not be less than 15° API.
- 4. Reservoir must be deep enough so that its pressure is greater than the miscibility pressure of CO2. Most of these limitations may be eliminated if steam is injected along with CO_2 .

In this experimental study these two most promising enhanced oil recovery processes, steam flooding and CO2 flooding, were combined to: (1) investigate the effect of CO2 on steam drive performance, (2) establish a CO₂/steam ratio which would maximize recovery.

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CHAPTER II

LITERATURE REVIEW

Steam flooding has been employed successfully to recover oil from medium to heavy oil reservoirs for the past three decades and has emerged as one of the most efficient processes in the field of Petroleum Engineering. Since its inception the steam drive process has been and is being studied in the laboratory, field and by the use of mathematical models. The ultimate objective of all this work is to develop a reliable engineering scheme to estimate oil recovery for a given set of conditions which could be altered to optimize steam flood design.

Willman et al²⁰ took the lead in conducting experimental research on linear laboratory cores when subjected to steam flooding and as a result concluded that:

- The recovery by steam injection is significantly higher than hot water flooding which in turn is more efficient than the conventional cold water flooding.
- The mechanisms responsible for the increased recovery are thermal expansion of oil, viscosity reduction and steam distillation with its related gas drive and solvent extraction effects.

They also suggested a procedure for estimating steam drive

performance based on the classical heat balance equations first presented by Marx and Langenheim²¹ assuming that the flow of heat from the steam zone into the hot liquid zone ahead of the condensation front is negligible (see fig. 2.1). This method of solution, in spite of its restrictive assumptions, has found considerable applications.

Lauwerier²² presented a model to predict the temperature distribution and thermal efficiency of a hot, non-condensable, fluid injection process by assuming thermal conductivities to be zero in the direction of flow and infinite over the longitudenal cross-section of the reservoir. Spillette²³ and Thomas²⁴ refined this model by relaxing the imposed restrictive assumptions and presented a numerical solution of the heat balance equations. While using these equations to describe the growth of steam zone, Mandl and Volek²⁵ discovered that the equations become inconsistent with the physical model of the steam drive process after a certain critical time which is dependent on the reservoir thickness, temperature and steam quality. They also found that the process of heat flow across the condensation front changes from purely conductive in nature to increasingly convective after the critical time. As a result of their findings they modified the existing equations and presented a method to determine the saturation at the downstream side of the condensation front. Gottfried²⁶ presented a theory of thermal recovery processes in linear systems and developed a sophisticated mathematical model that explicitly accounted for the conduction-convection heat transfer with convective external heat loss, aqueous phase change and hydrodynamics

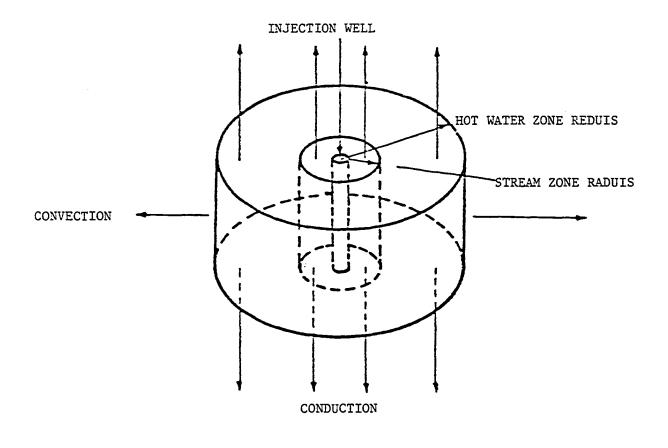


Figure 2.1: Willman et al.²⁰ Steam Drive Model

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of three phase flow. Although considerable computer time is required to obtain solutions of the system of equations to predict the temperature, pressure and saturation profiles in space and time, the model represented a major advance in ability to simulate the physical & chemical phenomena observed in thermal recovery experiments.

Farouq Ali²⁷ used the Marx and Langenheim approach to determine the effects of changes in the thermal properties of the overburden and underburden on steam flood performance and presented estimates of the error caused by the usual assumption of identical properties for overlying and underlying formations. In another related investigation²⁸ he studied the effects of variable rates of steam injection on the extent of heated area of the reservoir.

Closmann²⁹ using the same approach studied the growth of steam zones as a function of time due to steam injection into stratified formation consisting of highly permeable paths of equal steam injectivity separated by impermeable layers of equal thicknesses. He found that the presence of more permeable stringers is beneficial to overall heating of the reservoir.

Baker^{30,31} conducted laboratory steam flood experiments using a radial flow model (see Fig. 2.2) that comprised of disc shaped sand-pack reservoir, and an overburden and underburden consisting of water saturated sand. He observed, among other things, significant gravity override at all injection rates and found that:

 The heat lost to the overburden and underburden when expressed as a fraction of total heat injected, is independent of injection rate and is solely a function

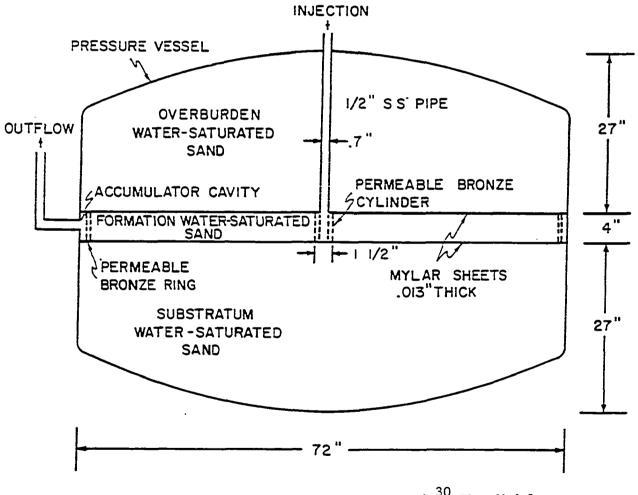


Figure 2.2: Schematic Drawing of Baker's ³⁰ Flow Model.

of time for a given formation thickness.

- 2. The division of heat between the steam and hot water zones is dependent on mass injection rate.
- 3. Gravity override is a strong function of injection rate and has minimal dependence on pressure and time. Gravity override has also been noticed by Blevins, Aseltine and Kirk³² while analysing a steam flood field project.

Shutler^{33,34} presented three phase steam flooding models for both linear and two dimensional fluid flow for the first time. The models allowed for interphase mass transfer between water and gas phases but assumed the oil to be non volatile and the hydrocarbon gas insoluble in liquid phases, thereby excluding the effects of miscibility from the steam drive process. Abdalla and coats³⁵ also derived a three phase, two dimensional steam flooding model and used an implicit pressure explicit saturation numerical technique to solve the system of equations. Shutler and Boberg³⁶ developed an analytic technique to calculate the size of the steam zone and to predict oil recoveries in one dimensional reservoirs using buckley Leverette method. The method is restricted to be used only for thin reservoirs.

Vinsome,³⁷ Coats et al³⁸, Coats³⁹ and Weinstein et al⁴⁰ advanced the technology a step further by presenting three phase, three dimensional steam flooding models. The advantages of these models over the previous ones stem from their ability to give simultaneous solution of the mass and energy balance equations; and implicit treatment of capillary pressure, water transmissibility,

and rates. The last two models also account for steam distillation.

Based on observations in Kern River steam drive project, Neuman⁴¹ included the important effects of gravity over-ride in his analytical model to describe the performance of a steam flood process in three dimensional reservoirs. He used Marx and Langenheim²¹ approach to calculate the rate of areal growth by the thermal balance between the net heat injection as steam and that required for steam condensation to sustain vertical growth. He predicted that the oil produced from the heated zone is a function of the net heat injected as steam.

Miller⁴² studied the effect of heat transport near the front and expansion or contraction due to thermodynamic phase change or chemical reaction at the front and showed that both these effects act to stabilize a moving front at which steam condenses and displaces water thus causing significant improvement in oil recovery.

Van Lookeren⁴³ following a different approach than Neuman, based on segregated flow principles and Dupuit⁴⁴ approximation, developed an analytical model to estimate the approximate shape of the steam/liquid interface for linear and radial flow systems. Rhee and Doscher⁴⁵ included the effect of steam distillation and extended this work to develop a method based on Higgen's Leighton's⁴⁶ areal model to determine the shape and growth of steam and hot condensate zones by integrating Van-Looeren's solution according to either Marx-Langenheim or Mandl Volek's approaches. One should not ignore the very interesting study of Myhill and Stegmeier⁴⁷ in the development of a prediction model based on simple energy balance equations to estimate ultimate oil/steam

ratios by assuming no contributions of the condensate zone to the oil recovery. They claimed that the model compared very well with the field and laboratory results. One should also mention the remarkable work of Ferrer et al⁴⁸ in deriving a three phase, three dimensional multicomponent flow model, designed to simulate steam injection processes. The model allows interphase mass and heat transfer to account for changes in oil composition in space and time.

Gomma⁴⁹ reported a noval curve matching model based on parametric studies done with a numerical simulator to predict steam flood performance.

Yortsos and Gavalas⁵⁰ reported analytical models that address the problem of heat transfer in detail in the hot liquid zone. They followed an integral balance approach to obtain upper bounds for one or multi-dimensional reservoirs under constant or variable injection rates and develope approximate asymptotic solutions in one dimensional reservoirs at constant injection rates. During the course of this work, they developed and delineated the range of validity of the existing models of Lauwrier, Marx-Langenheim, Mandl-Volek, Myhill-Stegmeier and Van Lookeren. Yortsos⁵¹ later extended this model to describe the fluid flow and the resulting saturation distributions inside the steam zone in a one dimensional steam injection precess.

Based on Van Lookeren and Myhill-Stegmeier Works, Jeff Jones⁵² proposed a steam drive model that can be used on a hand-held programmable calculator. The model comprises two integrated components. The first component calculates an optimal steam rate for

a given set of steam and reservoir parameters, while the second component calculates the oil production history by using the data obtained in the first component. He claimed that the results obtained by using this model matched well with the field resutls.

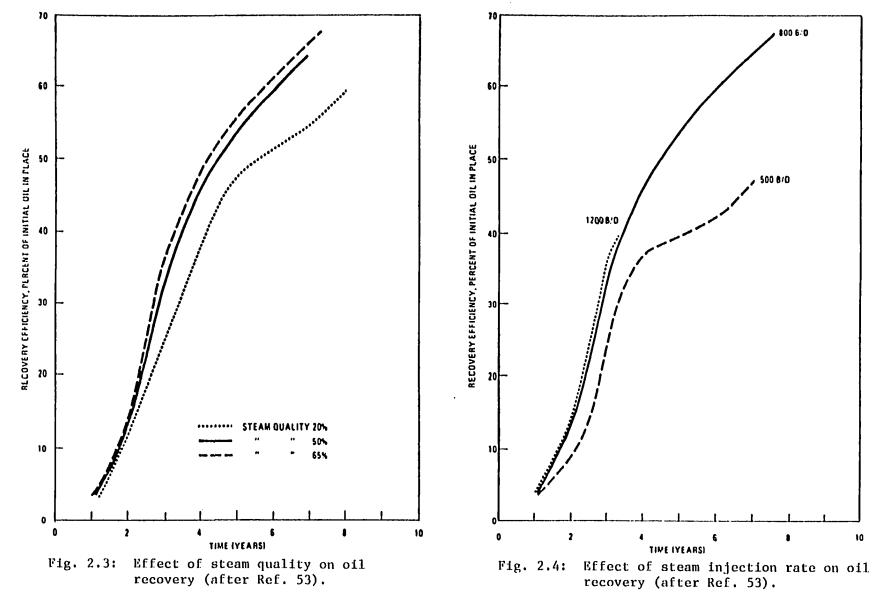
Moughamian et al⁵³ studied the effects of selected reservoir and operating parameters on oil recovery. They found among other things, that reservoir dip, steam quality, and steam injection rate are among the most important parameters affecting recovery efficiency (see Figures 2.3 through 2.7).

Krueger⁵⁴ extended Miller's theory to include injection of Nitrogen, a non condensing gas, together with the steam to study the stability of a flat condensation front displacing water. He concluded among other things that:

- Injection of Nitrogen together with steam increases.
 the possibility of having fingers compared with no Nitrogen case.
- Cooling fingers and surface tension have a stabilizing effect.
- 3. Increasing temperature has a destabilizing effect.

Closmann and Seba⁵⁵ conducted an experimental study to determine oil recovery by steam injection in linear systems and to investigate the effects of core length, saturation variations, pore size, and injection rates on residual oil saturations. They found among other things that:

 The breakthrough oil saturation is dependent on oil/water viscosity ratio evaluated at steam temperature and is not influenced by core length.



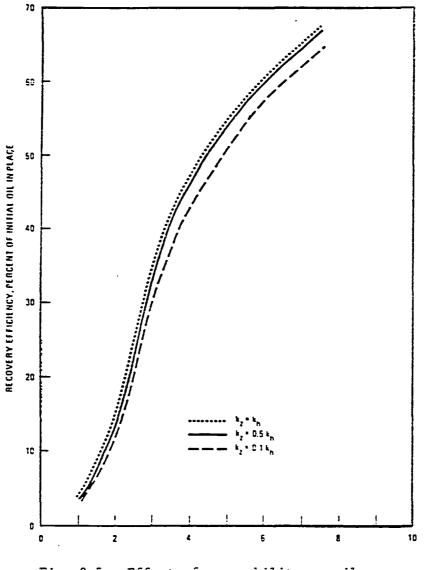


Fig. 2.5: Effect of permeability on oil recovery (after Ref. 53).

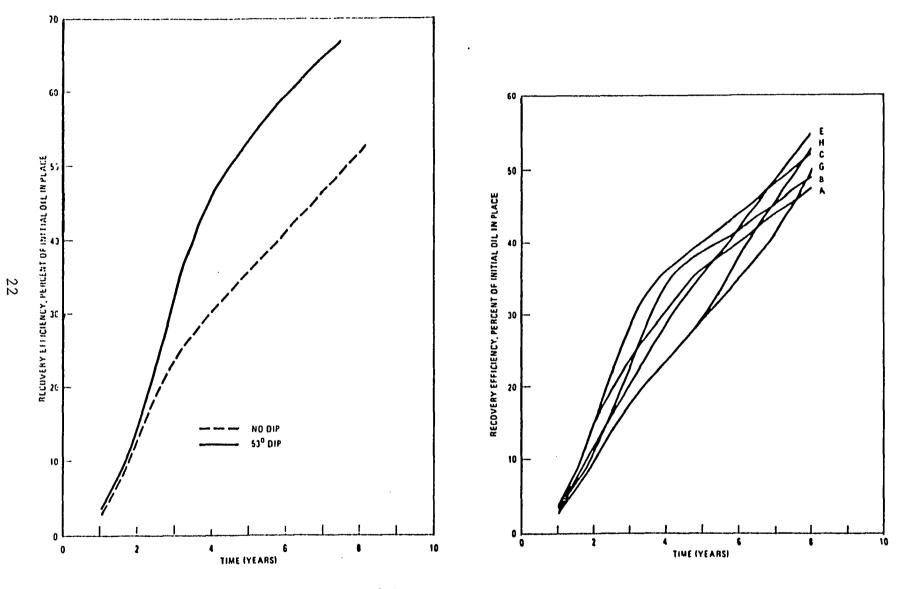


Fig. 2.6: Effect of dip on oil recovery (after Ref. 53).

Fig. 2.7: Effect of well location on oil recovery (after Ref. 53).

 The oil produced after steam breakthrough is also a function of oil/water viscosity ratio at steam temperature.

Steam injection technology has advanced significantly since its inception by the continuous research and analysis of laboratory and field results thus providing greater understanding of the process and its mechanisms. Several investigators have tried to improve the already good performance of the steamflooding process by the use of additives such as alkalis, polymers, solvents, surfactants etc. with steam but no systematic evaluation was either presented or published. In this connection Leung⁵⁶ applied numerical techniques to evaluate the effect of simultaneous steam and carbon dioxide injection on the recovery of heavy oil and found that the addition of carbon dioxide to injected steam improves the ultimate recovery slightly but enhances the oil production rate significantly before the steam breakthrough.

Mechanisms:

Steam flooding is a complex oil displacement proces that defies an exact description. As soon as steam enters the formation, it starts rapidly migrating upward due to strong gravitational gradients while advancing into the originally cool reservoir. As the steam zone grows, gravity overlay occurs, and this overlay increases as steam injection progresses. Simultaneously, part of the steam condenses forming a bank of water and displaced oil. The reservoir may then be divided into two distinct zones separated by a moving boundary: 1.) steam zone 2.) a hot

liquid zone which also includes initial cold region (see Fig.2.8).

The principal mechanisms responsible for the enhanced oil recovery are identified as:

- 1. Steam drive
- 2. In Situ Solvent drive
- 3. Viscosity reduction
- 4. Thermal permeability and capillary pressure variations
- 5. Thermal expansion
- 6. Gravity Segregation
- 7. Solution gas drive
- 8. Emulsification

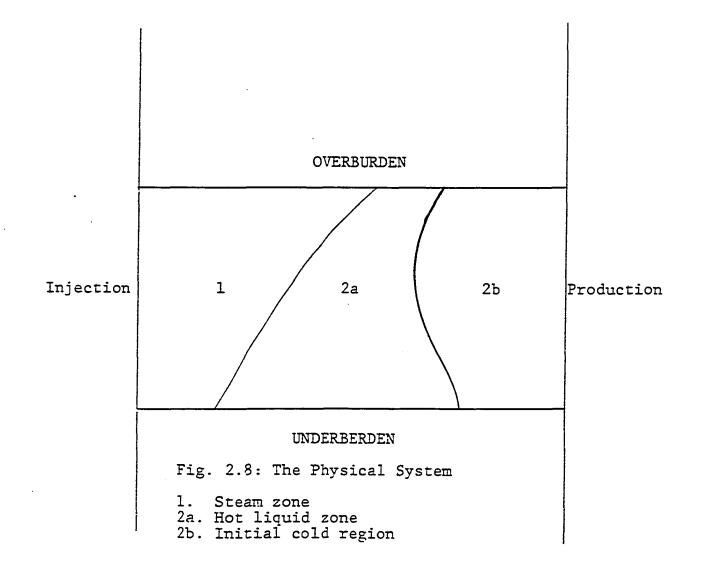
 ${\rm WU}^{57}$ discussed these mechanisms in detail.

STEAM DRIVE AND IN-SITU SOLVENT DRIVE

Steam distillation and steam displacement are the two important mechanisms known to exist in the steam zone. A fraction of the crude oil in the steam zone vaporizes and is carried forward through the advancing steam. These hydrocarbon vapors condense along with steam, mixing with the original crude at the condensation front to form a hot water zone and a hydrocarbon distillate or solvent bank. The distillate bank drives the oil miscibly ahead of the front followed by the steam drive which eventually establishes a low residual oil saturation in the steam zone.

VISCOSITY REDUCTION AND THERMAL EXPANSION

In addition to the solvent dilution, the most important mechanisms in the hot condensate zone responsible for enhanced



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oil production rate, are viscosity reduction and thermal expansion. Fast heating by condensing steam raises the temperature resulting in thermal expansion or swelling of the oil and a significant reduction in its viscosity. The viscosity reduces to such an extent that the condensed hot water is able to displace the heated oil relatively efficiently. Thermal expansion increases the oil saturation and decreases its density thus resulting in an increased relative permeability to oil.

THERMAL PERMEABILITY AND CAPILLARY PRESSURE VARIATIONS

Several investigators^{58,59,60,61} have published results showing variations in relative permeability and capillary pressure due to changes in temperature. All these workers found that with an increase in temperature:

- 1. The irreducible water saturation increased while the residual oil saturation decreased significantly.
- 2. The relative permeability curve shifted in the direction of increasing water saturation suggesting an increase in relative permeability to oil for a given water saturation.
- 3. The relative permeability ratio decreases.

They attributed these changes to the changes in rock-fluid inter-action or wettability. Since wettability is characterized by contact angle, they investigated the effect of temperature and found that the contact angle decreased with an increase in temperature, indicating that the system becomes more water wet with increasing temperature.

Davidson⁶¹ presented data showing a decrease in oil-water interfacial tension with increasing temperature. Figures 2.9 through 2.12 show these results. The overwhelming evidence suggests that variations in relative permeability and capillary pressure with increasing temperature are important recovery mechanisms in steam drive.

GRAVITY SEGREGATION AND EMULSIFICATION

As steam is injected, it channels through the reservoir and because of gravity its fingers rise to the top of the permeable sand. The fingers then spread out and after the overlay has occured, the principal forces causing the oil flow are the gravity head and steam drag. Steam sweeps or drags the underlying oil towards the producing well. Hot water falls out of the steam zone due to gravity as it condenses, and establishes a hot water displacement below the interface. Thus gravity override and the underrunning of hot water play an important role as the displacement mechanisms in the hot condensate zone.

It has also been suggested by a number of investigators and there is significant evidence that emulsification of the oil by the condensing high velocity steam is an important factor contributing to the mobilization of the heated oil.

K.C. Hong and J.W. Ault⁶⁶ studied the effects of noncondensable gas injection on oil recovery by steamflooding and reported among other things that the injection of a noncondensable gas with steam significantly accelerates oil production.

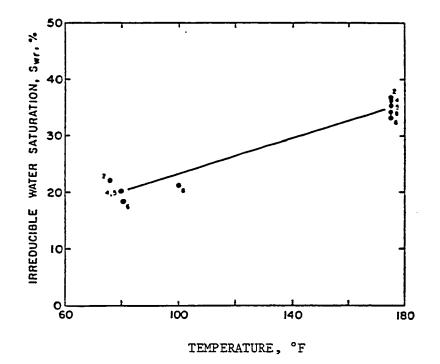


Figure 2.9: Effect of temperature on irreducible water saturation (after Ref. 61).

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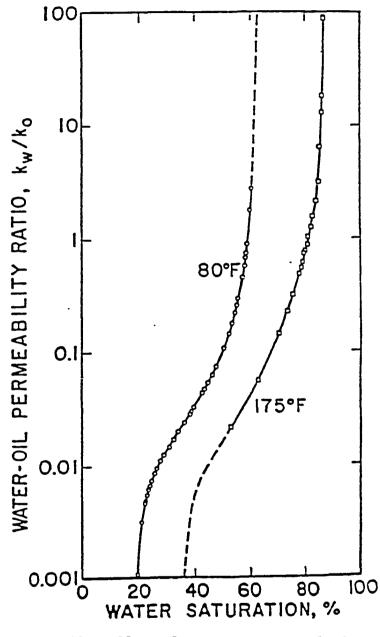
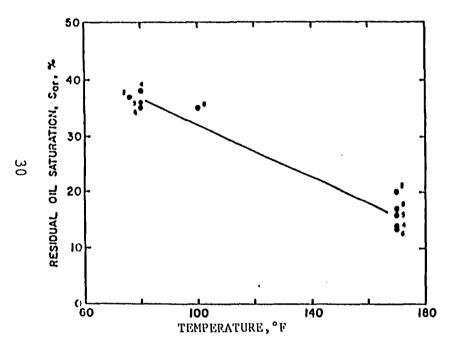


Fig. 2.10: Effect of temperature on relative permeability ratio (after Ref. 61).



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Fig. 2.11: Effect of temperature on residual oil saturation (after Ref. 61).

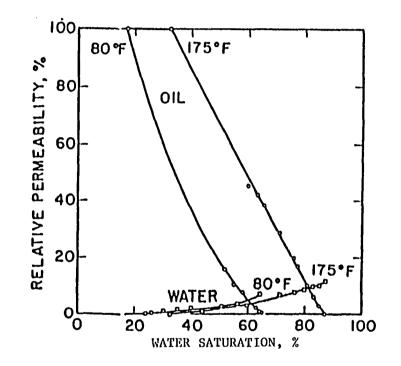


Fig. 2.12: Individual relative permeability as a function of temperature (after Ref. 61).

CHAPTER III

MATHEMATICAL FORMULATION

A mathematical model is nothing but a set of equations describing certain physical processes occuring in the reservoir. These equations express conservation of some quantity flowing through the reservoir. This model consists of equations expressing conservation of energy; conservation of mass for each component, phase equilibrium relationships and algebraic constraints. A general energy balance equation can be derived expressing conservation of the flowing quantity of interest.

Consider a small element of reservoir space $\Delta X \ \Delta Y \ \Delta Z$, shown in figure 3-1. The element is a rock containing fluids in the pore space.

A balance about the element, expressing conservation of the flowing thermal energy over a small time increment Δt , is given as:

Where

Energy in = Amount of the energy flowing into the element during time Δt .

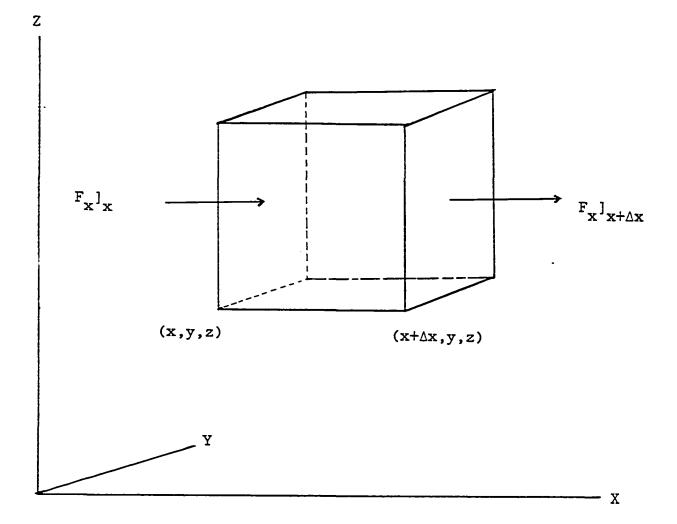


Fig. 3.1 SYSTEM FOR MASS OR HEAT BALANCE

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Energy out = Amount of energy flowing out of the element during time *it*.

Gain in internal energy = Amount of energy in the element at time "t+ Δ t" - Amount present at time"t".

The total energy flux due to flow of a fluid in the x-direction is the sum of the conductive and convective components (radiation neglected).

$$(F_{e,x}) = F_{k,x} + F_{c,x}$$
(3.2)

Where

 $(F_{e,x}) = Total energy flux in the x-direction$

F_{k,x} = Conductive heat flux in the x-direction i.e; the rate of heat transfer by conduction in the positive x-direction per unit cross-sec-tional area normal to the x-direction.

 $F_{c,x}$ = Convective heat flux in the x-direction.

$$F_{k,x} = -k_h - \frac{\partial T}{\partial X}$$
 (3.3)

Where

- k = Thermal Conductivity
- $U_x = x$ -component of the volumetric flux.
- h_{f} = Heat content of the fluid

Referring to figure 3.1, flow into the element takes place at three faces of areas:

$\Delta \mathbf{Y} \Delta \mathbf{Z}$	at	position X
$\Delta X \Delta Z$	at	position Y
ΔΧ ΔΥ	at	position Z

Therefore

Energy in = $[(F_x)_x \Delta Y \Delta Z + (F_y)_y \Delta x \Delta Z + (F_z)_z \Delta x \Delta Y] \Delta t...(3.5)$

Where

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 ${\bf F}_{\bf x}, \ {\bf F}_{\bf Y}, \ {\bf F}_{\bf Z}$ are fluxes at the three faces mentioned earlier.

Similarly flow out of the element takes place at three faces of areas

$\Delta \mathbf{Y} \Delta \mathbf{Z}$	at	position	$\mathbf{x} + \Delta \mathbf{x}$
$\Delta X \Delta Z$	at	position	$\mathbf{X} + \mathbf{A}\mathbf{X}$
$\Delta X \Delta Y$	at	position	$Z + \Delta Z$

Therefore

Energy out =
$$[(F_x)_{x+\Delta x} \quad \Delta y \quad \Delta z + (F_y)_{y+\Delta y} \quad \Delta x \quad \Delta z + (F_z)_{z+\Delta z} \Delta x \Delta y] \Delta t$$
.
3.6

Q = Rate of energy input per unit volume of the element. And finally "gain" during time Δt

Gain in internal energy = $[(\rho e)_{t+\Delta t} - (\rho e)_{t}] \Delta x \Delta y \Delta z \dots 3.8$

Where n_p $\rho e = (1-\phi) M_c \Delta T + \phi \sum_{i=1}^{2} S_i \rho_i e_i$

substitution of equations 3.5 through 3.8 in 3.1 yields:

$$-[(F_{x}) - (F_{x})] \Delta y \Delta z \Delta t [(F_{y}) - (F_{y})] \Delta x \Delta z \Delta t - [(F_{z}) - (F_{z})] \Delta x \Delta y$$

Dividing each term in equation 3.9 by $\Delta x \Delta y \Delta z \Delta t$, gives

$$\frac{(\mathbf{F}_{\mathbf{x}}) - (\mathbf{F}_{\mathbf{x}})}{\Delta \mathbf{x}} - \frac{(\mathbf{F}_{\mathbf{y}})}{\Delta \mathbf{y}} - \frac{(\mathbf{F}_{\mathbf{y}})}{\nabla \mathbf{y}} - \frac{(\mathbf{F}_{\mathbf{z}})}{(\mathbf{F}_{\mathbf{z}}) - (\mathbf{F}_{\mathbf{z}})} - \frac{(\mathbf{F}_{\mathbf{z}})}{\Delta \mathbf{z}} + \dot{\mathbf{Q}}$$

$$= \frac{(pe)_{t+\Delta t} - (pe)_{t}}{\Delta t}$$

Taking limits, when $\Delta x, \Delta y, \Delta z$, and Δt tend to zero

Continuity Equation:

Applying law of conservation of mass for each component-----oil,water, steam, hydrocarbon gas and CO₂: mass in --- mass out + mass input from sources

Let the mole fraction of any component j in the gas, oil and water phases be denoted by y_j , x_{oj} , and x_{wj} respectively: At equilibrium conditions

From equations 3.12 & 3.13

Also from Daltons law:

Where

- p_s = Saturation pressure of steam which is also partial
 pressure of steam
- $p_g = Pressure of the gas phase$

Also

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$$(F_{m,x})_{j} = M_{j} \left[\frac{\overset{U}{g,x} \overset{\circ}{g,y}_{j}}{\overset{M}{g}} + \frac{\overset{U}{o,x} \overset{\circ}{o} \overset{x}{oj}}{\overset{M}{o}} + \frac{\overset{U}{w,x} \overset{\circ}{x} \overset{x}{wj}}{\overset{M}{w}} \right]$$

Where

$$c_{j} = \text{concentration of component } j \text{ in terms of } y_{j}$$
$$= \frac{M_{j} \rho_{g} y_{j}}{M_{g}}$$

An approach similar to the one used for deriving energy balance equation, yields:

Accumulation of component j

$$= \frac{\partial}{\partial t} \left[M_{j} \phi \left(\frac{\rho_{g} S_{g} y_{j}}{M_{g}} + \frac{\rho_{o} S_{o} x_{oj}}{M_{o}} + \frac{\rho_{w} S_{w} x_{wj}}{M_{w}} \right]$$

Also from Darcy's law:

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Substitution of equations 3.18 through 3.23 in equation 3.11 yields the differential equations describing the law of conservation of mass for each component j.

$$- \left[\frac{\partial}{\partial x} (F_{m,x})_{j} + \frac{\partial}{\partial y} (F_{m,y})_{j} + \frac{\partial}{\partial z} (F_{m,z})_{j}\right] + m_{j}$$

$$= \frac{\partial}{\partial t} \left[M_{j} \phi \left(\frac{\rho_{g} S_{g} y_{j}}{M_{g}} + \frac{\rho_{o} S_{o} x_{oj}}{M_{o}} + \frac{\rho_{w} S_{w} x_{wj}}{M_{w}}\right)\right] \dots 3.24$$
or

$$- \frac{\partial}{\partial x} M_{j} \left(\frac{x_{0j} \rho_{0}}{M_{0}} U_{0,x} + y_{j} \frac{\rho_{g}}{M_{g}} U_{g,x} + x_{wj} - \frac{\rho_{w}}{M_{w}} U_{w,x} \right)$$

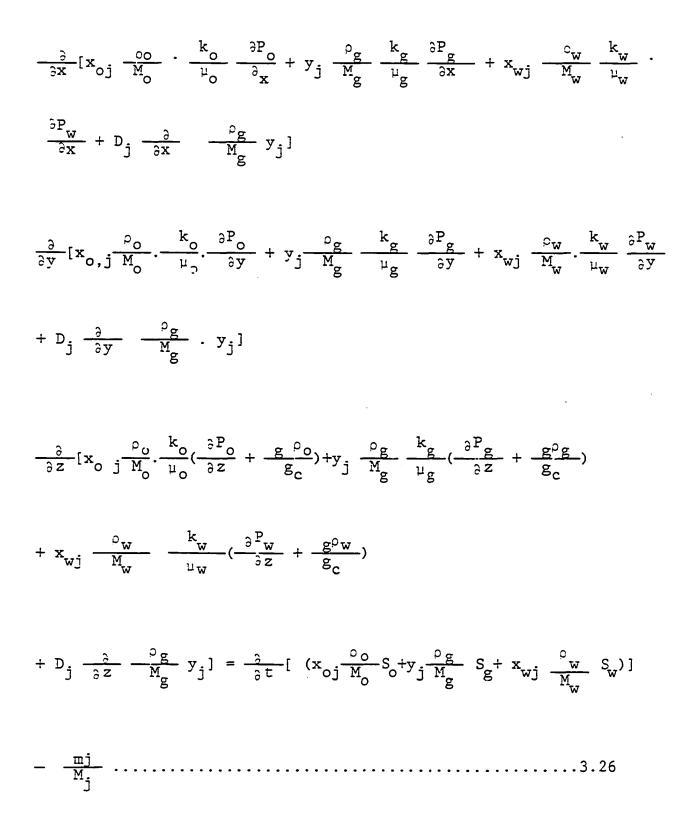
$$- D_{j} \frac{\partial}{\partial x} \frac{\rho_{g} y_{j}}{M_{g}}$$

$$- \frac{\partial}{\partial y} M_{j} \left[x_{0j} \frac{\rho_{0}}{M_{0}} U_{0,y} + y_{j} - \frac{\rho_{g}}{M_{g}} U_{g,y} + x_{w,j} - \frac{\rho_{w}}{M_{w}} U_{w,y} - D_{j} \frac{\partial}{\partial y} \frac{\rho_{g} y_{j}}{M_{g}} \right]$$

$$- \frac{\partial}{\partial z} M_{j} \left[x_{0j} \frac{\rho_{0}}{M_{0}} U_{0,z} + y_{j} - \frac{\rho_{g}}{M_{g}} U_{g,z} + x_{wj} \frac{\rho_{w}}{M_{w}} U_{w,z} - D_{j} - \frac{\partial}{\partial z} - \frac{\rho_{g} y_{j}}{M_{g}} \right]$$

$$+ m_{j} = \frac{\partial}{\partial t} \left[M_{j} \phi(x_{0j} - \frac{\rho_{0}}{M_{0}} S_{0} + y_{j} - \frac{\rho_{g}}{M_{g}} S_{g} + x_{wj} - \frac{\rho_{w}}{M_{w}} S_{w} \right] \dots 3.25$$

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or

For a reservoir with n_c components, there will be n_c differential equations similar to equation 3.26 (one for each component), one energy balance equation (3.10) and 2 n_c equilibrium equations such as equations (3.12 through 3.14).

Therefore:

- Total number of equation available = $3 n_c + 1$ Total number of unknowns = $3 n_c + 7$
- i. $3 n_{c} (y_{j}, x_{oj} \text{ and } x_{wj})$ in each continuity equation
- ii. Temperature
- iii. Pressures of each phase (p_w, p_g, p_o)
- iv. Saturations of each phase (S_w, S_c, S_c)

Therefore six additional equations are required to determine all the unknowns and these are:

We have:

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number of phases $n_p = 3$ (Gas, Water, & Oil) number of component $n_c = 3$ (CO₂ that can exist in both the gas phase and the oil phase, water, oil).

PHASE

Componenets	<u>i = g (gas)</u>	<u>i = o (oil)</u>	<u>i = w (water)</u>
J=1 (CO ₂₎	y ₁	^x ol	x _{wl} = o
J=2 (water)	y ₂ Ps Pg	x _{ow} = o	x _{ww} = 1
J=3 (oil)	^y 3 = 0	^x o3	$x_{w3} = 0$
3 ∑ j=1	1	l	1

For dead oil, there is no partitioning into or out of the oil phase. Therefore, the equation expressing conservation of mass applied to the oil phase, as obtained from equation 3.25, is given by

$$= \frac{\partial}{\partial x} \left[M_{0} \frac{x_{00}^{\circ} \circ}{M_{0}} \cdot U_{0,x} \right] - \frac{\partial}{\partial y} \left[M_{0} \frac{x_{00}^{\circ} \circ}{M_{0}} U_{0,y} \right] - \frac{\partial}{\partial z} \left[M_{0} \frac{x_{00}^{\circ} \circ}{M_{0}} U_{0,z} \right]$$

$$= \frac{\partial}{\partial t} \left[M_{0} \phi (x_{00} \frac{\rho_{0}}{M_{0}} S_{0}) \right]$$
or

$$-\frac{\partial}{\partial x}(\circ_{0} U_{0,x}) - \frac{\partial}{\partial y}(\rho_{0} U_{0,y}) - \frac{\partial}{\partial z}(\rho_{0} U_{0,z}) = \phi \frac{\partial}{\partial t}(\rho_{0} S_{0})$$

or
$$\frac{2}{2t} (c_0 S_0) + \nabla \cdot (\rho_0 U_0) = 0 \dots 3.38$$

.

Similarly the continuity equation for water, in both the liquid and gaseous phases, is given by

$$\phi \frac{\partial}{\partial t} (c_w S_w) + \nabla \cdot (c_w U_w) + \phi \frac{\partial}{\partial t} (c_s S_s) + \nabla \cdot (c_s U_s) = 0$$

where the subscripts o, w, and s denote oil, water and steam respectively.

Conservation of energy for the reservoir containing dead oil is given by equation 3.10, which can also by written as:

$$-\frac{\partial}{\partial \mathbf{x}}\left[-\mathbf{k}_{\mathrm{h}}\frac{\partial \mathrm{T}}{\partial \mathbf{x}}+\mathbf{U}_{\mathrm{x}}\mathbf{p}_{\mathrm{f}}\mathbf{h}_{\mathrm{f}}\right]-\frac{\partial}{\partial \mathrm{y}}\left[-\mathbf{k}_{\mathrm{h}}\frac{\partial \mathrm{T}}{\partial \mathrm{y}}+\mathbf{U}_{\mathrm{y}}\mathbf{p}_{\mathrm{f}}\mathbf{h}_{\mathrm{f}}\right]$$

$$-\frac{\partial}{\partial z}[k_{h} \frac{\partial T}{\partial z} + U_{z} \circ_{f} h_{f}] = -\frac{\partial}{\partial t}[(1-\phi) \circ_{r} C_{r}T + \phi \Sigma S_{i} \circ_{i} e_{i}]$$

or

$$[(1-\phi)\circ_{r}C_{r} + \phi(\circ_{o}C_{o}S_{o} + \circ_{w}C_{w}S_{w})] - \frac{\partial T}{\partial t} + L_{v}[\phi \frac{\partial (\circ_{s}S_{s})}{\partial t} + \nabla \cdot \circ_{s}U_{s}]$$

+
$$(\circ_{o} C_{o} U_{o} + \circ_{w} C_{w} U_{w})$$
. $\forall T + \circ_{s} U_{s} \cdot \forall h_{s} + \forall \cdot (-k_{h} \forall T) = 0 \dots 3.40$

For only steam injection, the saturation identity becomes

At saturated condition,

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 $P_{sat} = P_{sat} (T_{sat.}) \dots Clausius-Clapeyron relationship....3.42$

The following equations, expressed in the functional form, describe the dependence of material properties on the thermodynamic state variables:

$\phi = \phi(\mathbf{x}, \mathbf{y}, \mathbf{z}) \dots$	
$\mu_{j} = \mu_{j} (T) \dots$	
k = k (x,y,z)	
k _{rj} = k _{rj} (S)	
$\rho_j = \rho_j (P,T) \ldots$	
$h_{s} = h_{s} (T) \dots$	
$L_v = L_v (T) \dots$	
$k_h = k_h (x,y,z) \dots$	
$\rho_r = \rho_r (x,y,z) \dots$	

For the cap and base rock, the energy equation can be written as:

$$P_{c} C_{c} \frac{\partial T}{\partial t} + \nabla (-k_{h} \nabla T) = 0 \dots 3.52$$

The Initial and Boundary Conditions:

The initial condition for the system are given by:

$$P(i,x,y,z) = P_i(x,y,z) \dots 3.54$$

Where the subscript i stands for the initial value. The mass flux is zero at all the boundaries with the exception of the wells, thus:

$$[o_j U_{jn}]^{1b} = [o_j U_{jn}]^{ub} = 0 \dots 3.56$$

where

lb = lateral boundary

ub = upper and lower boundary.

subscript n refers to the flow in a direction normal to the boundary.

Similarly, the convective heat flux is zero at all the lateral boundaries with the exception of the wells; thus

At the upper and lower boundaries

Mass injection rate of wet steam at the injection well is given by:

At the production well, the bottom hole pressure is given by: $P = P_p(t) \dots 3.60$

The rate of energy injection at the injection well :

Where

 $f_s = quality of the steam$ $\Delta T = T_s - T_i$

DIMENSIONAL ANALYSIS:

This study was devoted to an experimental determination of oil recovery by simultaneous injection of carbon dioxide and steam in linear systems. The main objective was in obtaining an overall view of the important variables that affected recovery. But for the sake of completeness dimensional analysis are performed to asses the practical value of scaling laws. The principal advantage of using dimensionless groupings is to reduce the number of independent variables in a problem.

Greetsma et al⁶² and Van Daalen and Domselaar⁶³ presented dimensionally scaled models of oil reservoirs under isothermal and nonisothermal conditions of water flooding, while Niko and Troost⁶⁴ developed a partial list of dimensionless parameters and appropriate set of scaling rules in the steam injection area.

The first step in deriving the important groups of parameters that are related to the steam injection processes is the development of scaling parameters which are obtained by either using Buckingham's π theorm or by inspectional analysis. Generally, when two physical systems are identical and are behaving similarly, known parameters of the one can be used to calculate the unknown parameters of the other. The details of these methods and a compilation of the most important dimensionless groups

and an example illustrating the procedure of converting prototype values to model values and vice versa are presented.

Buckingham's π theorm:

This theorm states that the number of independent dimensionless groups is equal to the difference between the number of physical variables and the number of basic dimensions used to express them. Dimensional constants are also included as variables. Designating dimensionless groups by the letters $\pi_1, \pi_2, \pi_3, \ldots$, the complete physical statement can be expressed in a functional form as

$$f(\pi_1, \pi_2, \pi_3 \dots) = 0$$

Let us consider the following seven variables

$$U_{c} = \text{Convective heat transfer coefficient } [Mt^{-3} T^{-1}]$$

$$k_{h} = \text{thermal conductivity } [M L t^{-3} T^{-1}]$$

$$L = \text{characteristic length } [L]$$

$$\rho = \text{density } [M L^{-3}]$$

$$u = \text{velocity } [L t^{-1}]$$

$$\mu = \text{viscosity } [M L^{-1} t^{-1}]$$

$$C_{p} = \text{specific heat } [L^{2} t^{2} T^{-1}]$$

These have four basic dimensions of mass, length, time, and temperature denoted by M, L, t and T respectively.

Therefore, the number of independent dimen- = [number of physical] - [number of basic] sionless groups variables dimensions = 7 - 4 = 3

Determination of dimensionless groups:

The dimensionless groups may be determined by using the following procedure:

- i. List all the variables involved and their dimensions.
- ii. Find the basic dimensions
- iii. Select a number of repeating variables equal to the number of basic dimensions from the list of the variables.
 - iv. Determine the dimensionless groups by solving the dimensional equations set up by combining the variables selected in step (iii) and each of the other remaining variables in turn.

If actual relationship among different variables of a problem is not known, a relation of the following form can be assumed to determine the pertinent dimensionless groups.

Where

(i) q_1 , q_2 , q_3 , q_4 , q_5 , are pertinent variables in the problem

(ii) a, b, c, d, e, are unknown exponents

On substitution of the basic dimensions in terms of M, L, t, and T for each of the variables in equation 3.62, the sum of the exponents of each of these basic dimensions must result in zero; thus leading to a set of four simultaneous equations containing a, b, c, d and e as the unknowns. Any four of these five exponents can be solved in terms of the remaining one. Then, back-substitution in equation 3.62 will result in one independent dimensionless group.

The aforementioned procedure will now be illustrated. Consider a problem where the variables are velocity, V, characteristic length L, and gravitational acceleration, g. Then

 $V^{a} L_{c}^{b} g^{c} = \pi_{1}$, a dimensionless quantity substitution of dimensions for V, L, and g, yields

$$[Lt^{-1}]^{a} [L]^{b} [Lt^{-2}]^{c} = \pi_{1}$$

The basic dimensions are L and t, since M and T are not present in the variables.

Exponent of L: a + b + c = 0Exponent of t: -a - 2c = 0From the above two equations, we have

a = -2cb = c

Back substitution for a and b, yields

$$V^{-2c} L_c^c g^c = \pi_1$$

or

$$\frac{v^2}{L_c g} = \pi$$

In the above example, there were three variables and two basic dimensions, thus giving rise to one dimensionless group. <u>Inspectional analysis</u>: A more straight forward and reliable way to derive the dimensionless groups is by inspectional analysis. The first step towards this end is the development of scaling parameters which are obtained by making the available governing equations, describing fluid flow and heat transfer, dimensionless; followed by determining the similarity parameters by inspectional analysis. The similarity parameters thus determined are combined or modified to obtain the desired scaling parameters.

<u>Dimensionless form</u>: The governing equations of the steam injection processes contain a number of physical variables. Each of these variables is replaced by the product of a dimensionless variable and a reference value of the variable. For instance, a variable U, is divided by its reference value, U_R , to obtain the dimensionless ratio of the variable, U_D , such as

$$U_D = \underbrace{U}_R$$
 or $U = U_R U_D$ 3.63

To make equation 3.38 (conservation of mass equation) dimensionless: Let

$$\phi = \phi_{R} \phi_{D}, \phi_{o} = \rho_{oR} \phi_{oD}, S_{o} = S_{R} S_{omD}$$
$$U_{o} = U_{R} U_{oD}, \quad t = t_{R} t_{D}$$
$$dt = t_{R} dt_{D}, \quad dx = L_{R} dx_{D}$$
$$\nabla = \frac{1}{L_{R}} \nabla_{D}$$

Substituting the above in equation 3.38, we have

$${}^{\phi}R \stackrel{\phi}{=} \frac{{}^{\rho}OR \stackrel{S_R}{\longrightarrow}}{t_R} \cdot \frac{{}^{\partial}({}^{\rho}OD \stackrel{S_{OmD}}{\longrightarrow})}{{}^{\partial}t_D} + \frac{1}{L_R} \nabla_D \cdot ({}^{\rho}OR \stackrel{\rho}{\longrightarrow}{}^{OD} \cdot U_R \stackrel{U}{\longrightarrow}{}^{OD}) = 0$$

or

- - -

$${}^{\diamond}R {}^{\phi}D - \frac{{}^{\diamond}OR}{t_R} S_R \cdot \frac{\partial ({}^{\diamond}OD}{\partial t_D} S_{OMD})}{\partial t_D} + \frac{{}^{\diamond}OR}{L_R} V_D \cdot ({}^{\diamond}OD V_{OD}) = 0$$

Multiplication of the above equation by
$$(\frac{L_R}{U_R})$$
 yields

$$(\frac{\circ_{R} \quad S_{R} \quad L_{R}}{U_{R} \quad t_{R}}) \quad \stackrel{\phi}{\to} \quad \frac{\partial \quad (\circ_{OD} \quad S_{OmD})}{\partial t_{D}} + \nabla_{D} \quad (\circ_{OD} \quad U_{OD}) = 0...3.64$$

Where

S_{omD} = dimensionless movable oil saturation

Movable oil saturation is used to improve the match of the relative permeabilities, since the residual oil saturation and interstitial water saturation generally are not the same between model and the prototype.

Similarly, the conservation of mass equation for water (equation 3.39) in the dimensionless form is,

The dimensionless form of the Darcy's equation is obtained by making proper substitutions in equation (3.21) and is given by

$$U_{R} U_{jD} = - \frac{k_{R} k_{D} k_{rj}}{\mu_{R} \mu_{jD}} \cdot \frac{P_{R}}{L_{R}} \nabla D^{P} D$$

Or

$$(\frac{\overset{\mu}{R} \overset{U}{R} \overset{L}{R}}{\overset{k}{R} \overset{P}{R}}) U_{jD} = \frac{\overset{k}{D} \overset{k}{rj}}{\overset{\mu}{jD}} \nabla_{D} P_{D} \dots \dots 3.66$$

The energy equation (equation 3.40) is made dimensionless as follows:

To make the first set of brackets dimensionless, let

Then

$$\{(1-\phi) \circ_{r} C_{r} + \phi[\circ_{o} C_{o}(S_{R} S_{omD} + S_{oRs}) + \circ_{w} C_{w}(S_{R} S_{wmD} + S_{wc})\} = \frac{\partial T}{\partial t}$$

or

$$[(1-\phi) \rho_{r} C_{r}^{+} \phi (\rho_{o} C_{o} S_{ors}^{+} \rho_{w} C_{w} S_{wc}^{-})] \frac{\partial T}{\partial t} + \phi (\rho_{o} C_{o} S_{R} S_{omD}^{-} + \rho_{w} C_{w} S_{R} S_{wmD}^{-}) \frac{\partial T}{\partial t}$$

or

· - ·

$$\circ_{cR} C_{cR} [(1-\phi) \circ_{r} C_{r} + \phi(\circ_{o} C_{o} S_{ors} + \phi_{w} C_{w} S_{wc})]_{D} - \frac{T_{R}}{t_{R}} - \frac{\partial T_{D}}{\partial t_{D}}$$

$$+ \circ_{R} \circ_{D} \cdot S_{R} \cdot \circ_{R} \cdot C_{R} \circ_{OD} C_{OD} \circ_{OD} \circ_{OD} \circ_{WD} \circ_{WD} S_{WD} \circ_{WD} \circ_{WD} \circ_{WD} \circ_{WD} \circ_{WD} \circ_{WD} \circ_{WD} \circ_{U} \circ_{T_{R}} \circ_{T_{D}} \circ_$$

$$(\frac{\rho_{cR} C_{cR} T_{R}}{t_{R}})[(1-\phi) \rho_{r} C_{r}^{+} \phi(\rho_{o} C_{o} S_{ors}^{+} + \rho_{w} C_{w} S_{wc}^{-})] \frac{\partial T_{D}}{\partial t_{D}}$$

$$+(\underbrace{-\frac{\phi_{R}}{\sigma_{R}}}_{R} \underbrace{\sigma_{R}}_{r} \underbrace{\sigma_{R}}_{r} \underbrace{\sigma_{R}}_{r} \underbrace{\sigma_{D}}_{r} \underbrace{\sigma$$

The second set of brackets in the energy equation (equation 3.40) is made dimensionless as follows:

$${}^{\mathrm{L}}_{\mathrm{vR}} {}^{\mathrm{L}}_{\mathrm{vD}} {}^{\left[\phi_{\tilde{R}}\phi_{D}} - \frac{\rho_{R}}{t_{R}} {}^{S_{R}} - \frac{\partial(\rho_{\tilde{s}D}S_{sD})}{\partial t_{D}} + \frac{\rho_{R}}{t_{R}} {}^{U_{R}} {}^{\nabla}_{\mathrm{D}} \cdot \rho_{sD} {}^{U_{sD}} {}^{]}$$

or

- - -

$$(\frac{\phi_{R} \ S_{R} \ \rho_{R} \ L_{vR}}{t_{R}}) \ L_{vD} \ \phi_{D} \ \frac{\partial (\rho_{sD} \ S_{sD})}{\partial t_{D}} + (\frac{\rho_{R} \ U_{R} \ L_{vR}}{L_{R}}) \ L_{vD} \ \nabla_{D} \ \rho_{sD} U_{sD}$$

The third set of brackets in equation 3.40 is made dimensionless as follows:

$$({}^{\rho}R {}^{\rho}OD \cdot {}^{C}R {}^{C}OD \cdot {}^{U}R {}^{U}OD + {}^{\rho}R {}^{\rho}WD \cdot {}^{C}R {}^{C}WD \cdot {}^{U}R {}^{U}WD) \cdot \frac{{}^{T}R}{{}^{L}R} {}^{\nabla}D {}^{T}D$$

or

$$\frac{\rho_{R} C_{R} U_{R} T_{R}}{L_{R}} (\rho_{OD} C_{OD} U_{OD} + \rho_{wD} C_{wD} U_{wD}) \cdot \nabla_{D} T_{D} \dots \dots 3.70$$

The fourth and fifth sets of brackets in the energy equation (equation 3.40) are now made dimensionless as follows:

$$P_{s} U_{s} = \nabla (L + C_{w} T) - k_{h} \nabla^{2} T$$

or

$$\stackrel{\rho_{R}}{\xrightarrow{}} \stackrel{\rho_{SD}}{\xrightarrow{}} \stackrel{U_{R}}{\xrightarrow{}} \stackrel{U_{SD}}{\xrightarrow{}} \frac{1}{L_{R}} \stackrel{\nabla_{D}}{\xrightarrow{}} \stackrel{(L_{VR}}{\xrightarrow{}} \stackrel{L_{VD}}{\xrightarrow{}} + C_{R} C_{WD} \cdot T_{R} T_{D}$$

$$- k_{hR} k_{hD} \frac{1}{L_{R}^{2}} \stackrel{\nabla_{D}}{\xrightarrow{}} T_{R} T_{D}$$

or

- - - -

$$(\frac{\rho_{R} U_{R} L_{vR}}{L_{R}}) \rho_{sD} U_{sD} \nabla_{D} L_{vD} + (\frac{\rho_{R} U_{R} C_{R} T_{R}}{L_{R}}) \rho_{sD} U_{sD} \nabla_{D} C_{wD} T_{D}$$

Equations 3.68, 3.69, 3.70, and 3.71 are now combined and multiplied by $(\frac{t_R}{\phi_R \rho_R S_R C_R T_R})$ to obtain the final dimensionless form of the

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$$(\frac{\rho_{cR} C_{cR}}{\phi_{R} \rho_{R} S_{R} C_{R}})[(1-\phi)\rho_{r} C_{r} + \phi(\rho_{o} C_{o} S_{oRS} + \rho_{w} C_{w} S_{wc})]_{D} \frac{\partial T_{D}}{\partial t_{D}}$$

+
$$\phi_D$$
 ($\rho_{oD} C_{oD} S_{omD}$ + $\rho_{wD} C_{wD} S_{wmD}$) $\frac{\partial T_D}{\partial t_D}$

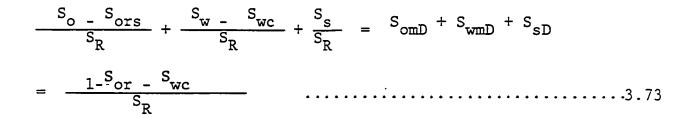
+
$$\left(\frac{L_{\mathbf{v}R}}{C_{\mathbf{r}}T_{\mathbf{R}}}\right)$$
 $L_{\mathbf{v}D}$. $\left[\phi_{\mathbf{D}} \frac{\partial (\rho_{\mathbf{s}D}S_{\mathbf{s}D})}{\partial t_{\mathbf{D}}} + \frac{U_{\mathbf{R}}t_{\mathbf{R}}}{\phi_{\mathbf{R}}S_{\mathbf{R}}L_{\mathbf{R}}}\nabla_{\mathbf{D}} \cdot \rho_{\mathbf{s}D}U_{\mathbf{s}D}\right]$

+
$$\left[\frac{U_R t_R}{\phi_R S_R L_R} (\frac{L_{vR}}{C_R T_R} + 1) \circ_{sD} U_{sD} \cdot \nabla_D (L_{vD} + C_{wD} T_D)\right]$$

+
$$\left(\frac{U_{R} + L_{R}}{\phi_{R} + S_{R} + L_{R}}\right) \left[\rho_{oD} + C_{oD} + \rho_{wD} + C_{wD} + U_{wD}\right] \cdot \nabla_{D} T_{D}$$

$$- \left(\frac{k_{hr} t_{R}}{\phi_{R} \rho_{R} S_{R} C_{R} L_{R}^{2}}\right) k_{hD} \tau_{D}^{2} T_{D} = 0 \dots 3.72$$

The dimensionless form of the movable saturations becomes:



Where

Subscript m represents movable.

Equations 3.42 through 3.51 are already in the dimensionless form.

The dimensionless form of the energy equation for the cap and base rock (equation 3.52) is given by:

$${}^{\rho}cR {}^{\rho}cD {}^{C}cR {}^{C}cD \frac{T_{R}}{t_{R}} \frac{\partial T_{D}}{\partial t_{D}} = k_{hR} k_{hcD} \cdot \frac{1}{L_{R}^{2}} \nabla_{D}^{2} T_{R} T_{D}$$

or

Equations 3.53 through 3.55 representing initial conditions are already in the dimensionless form.

The boundary conditions given by equations 3.56 and 3.57 are also dimensionless, but equation 3.58 is made dimensionless as follows:

$$\begin{bmatrix} k_{hR} & k_{hrD} & \frac{T_R}{n_R} & \frac{\partial T_D}{\partial n_D} \end{bmatrix}_r^{ub} = \begin{bmatrix} k_{hR} & k_{hcD} & \frac{T_R}{n_R} & \frac{\partial T_D}{\partial n_D} \end{bmatrix}_c^{ub}$$

or

The equation giving mass injection rate of wet steam at the injection well (equation 3.59), becomes in the dimensionless form as:

$$w_{R} w_{D} = \pi L_{R} d_{D} \int_{0}^{z_{D}} (\rho_{R} \rho_{wD} \cdot U_{R} U_{wnD} + \rho_{R} \rho_{sD} U_{R} U_{snD}) L_{R} \frac{dz_{D}}{\cos\theta}$$

or

$$(\frac{\mathbf{w}_{R}}{\mathbf{\rho}_{R} \mathbf{u}_{R} \mathbf{L}_{R}^{2}}) \mathbf{w}_{D} = \pi \mathbf{d}_{D} \int_{0}^{z} (\mathbf{\rho}_{wD} \mathbf{u}_{wnD} + \mathbf{\rho}_{sD} \mathbf{u}_{snD}) \frac{\mathbf{d}z_{D}}{\mathbf{cos}\theta}$$

Equation 3.60, giving bottom hole pressure at the production well remains unaffected in form and is dimensionless.

The equation for the energy injection (equation 3.61) in the dimensionless form becomes:

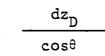
 $W_R W_D (f_{sR} f_{sD} L_{vR} L_{vD} + C_R C_{wD} T_R \Delta T_D)$

 $= \pi L_R d_D \int_0^{z_{tD}} [\rho_R \rho_{wD} U_R U_{wnD} C_R C_{wD} T_R \Delta T_D + \rho_R \rho_{sD} U_R U_{snD} h_R h_{sD}]$ $\cdot L_R \frac{dz_D}{\cos^2}$

or

$$W_R W_D [(f_{sR} L_{vR}) f_{sD} L_{vD} + (C_R T_R) C_{wD} \Delta T_D]$$

 $= \pi L_{R}^{2} d_{D} \int_{0}^{z} [(\rho_{R} U_{R} C_{R} T_{R}) \rho_{wD} U_{wnD} C_{wD} \Delta T_{D}^{+} (\rho_{R} U_{R} h_{R}) \rho_{sD} U_{snD} h_{sD}]$



•

or

$$(-\frac{w_R}{\rho_R U_R L_R^2}) w_D \left[\left(\frac{f_{sR} L_{vR}}{C_R T_R} \right) f_{sD} L_{vD} + C_{wD} \Delta T_D \right]$$

= $\pi d_D \int_{0}^{z} (\rho_{wD} U_{wnD} C_{wD} \Delta T_D + \left(\frac{h_R}{C_R T_R} \right) \rho_{sD} U_{snD} h_{sD} \right] \frac{dz_D}{cose}$

Determination of Independent Dimensionless Groups:

The dimensionless groups are simply the co-efficients of the individual terms of the governing equations, written in the dimensionless form, describing fluid flow and heat transfer. The independent dimensionless groups are, then, determined either by observation or by using Buckingham Pi theorm. Stegemeirs et al discussed this subject in detail and derived these groups listed in tables 3-1 and 3-2.

TABLE 3-1

Similarity	Parameters	For	Steam	Processes	(after	Ref.65)
Parameters					Source	Equation
$\frac{k_R P_R t_R}{R S_R M_R L_R^2}$					3.64	and 3.66
P _R P _R S _R L _R					3	.66
C _R T _R					3	.72
^k hR ^t R R ^S R R ^C R ^L R					. 3	.72
$\frac{k_{hR}}{cR} \frac{t_{R}}{c_{R}}$					3	.74
<u> </u>					3.66 a	nd 3.76
<u> </u>					3	.77

Parameter	Number
P _R ° _R g _R L _R	I
$(\frac{f_{sR} L_{vR}}{C_R T_R} + 1) A^*$	II
$\frac{f_{sR} \mu_{sR} \rho_{R}}{\mu_{R} \rho_{sR}}$	III
$\frac{k_{hR}}{\phi_R} c_R c_R L_R^2 \cdot A^*$	IV
$\frac{\phi_{R} s_{R} \mu_{R} L_{R}}{k_{R} \rho_{R} s_{R} t_{R}}$	V
$\frac{w_{R} t_{R}}{\sigma_{R} \phi_{R} S_{R} L_{R}^{3}}$	VI

* When $\phi \triangle s$ is not matched, A takes on a value between unity and

$$\circ_{R} \circ_{R} (\xrightarrow{\circ_{R}} C_{R}).$$

If reservoir heating or heat production predominates, use unity, if cap and base rock heating predominates, use $\phi_R = s_R \frac{c_R}{c_R} \frac{C_R}{c_R}$.

64

Scaling Parameters For Steam Processes (after Ref. 65)

Selection of the characteristic quantities:

The characteristic quantities are chosen in such a way that the system does not change when the scale is changed from the model to the prototype. For example the reference saturation, S_R , is chosen as

The saturation identity, given by equation 3.67, then becomes:

Equation 3.79 is equally good for both model and prototype if S_{ors} and S_{wc} are constant. Thus equation 3.53, for the initial condition on oil saturation, becomes:

$$\frac{S_{o}(i, x, y, z) - S_{ors}}{1 - S_{ors} - S_{wc}} = \frac{S_{oi}(x, y, z) - S_{ors}}{1 - S_{ors} - S_{wc}} \dots 3.80$$

and for the model and the prototype to be similar

$$\frac{[\frac{S_{oi}(x, y, z) - S_{ors}]}{1 - S_{ors} - S_{wc}}]_{prototype} = 1$$

rence pressure is chosen

$$p_R = p_{max} - p_{min} = \Delta p_{max} \dots 3.82$$

hus equation 3.54, the initial condition on pressure, becomes:

$$\frac{p(i, x, y, z) - p_{\min}}{p_{\max} - p_{\min}} = \frac{p_i(x, y, z) - p_{\min}}{p_{\max} - p_{\min}} \cdots \cdots 3.83$$

and for the model and prototype to be similar

$$\begin{bmatrix} p_{i}(x, y, z) - p_{min} \\ p_{max} - p_{min} \end{bmatrix} \text{prototype} \qquad = 1$$
$$\begin{bmatrix} p_{i}(x, y, z) - p_{min} \\ p_{max} - p_{min} \end{bmatrix} \text{model}$$

Example of Scaling Prodedure:

Scaling of pressure:

Given:

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Model well production pressure, $(P_p)_M = 61$ Psia

Prototype to model length ratio $\left[\frac{L_p}{L_M}\right] = \gamma(L) = 200$ Prototype to model density ratio $\left[\frac{(\rho_0)_p}{(\rho_0)_M}\right] = \gamma(\rho) = 1$

Using scaling parameter 1 in table F2, we have

$$\left(\frac{\Delta P}{\rho g L}\right)_{p} = \left(\frac{\Delta P}{\rho g L}\right)_{M}$$

$$\left[\frac{P - P_{p}}{\rho g L}\right]_{p} = \left[\frac{P - P_{p}}{\rho g L}\right]_{M}$$

$$\left(P - P_{p}\right)_{p} = \frac{(\rho g L)_{p}}{(\rho g L)_{M}} \left(P - P_{p}\right)_{M}$$

$$P_{p} = 200 \left(P - P_{p}\right)_{M} + \left(P_{p}\right)_{P}$$

$$P_{p} = 200 \left(P - 61\right)_{M} + 100 \dots 3.85$$

$$= 200 P_{m} - 12200 + 100$$

$$= 200 P_{m} - 12100$$

Where

••• • •• • •

 P_{M} = model pressure P_{p} = prototype pressure If

Then

Prototype steam pressure =
$$200(67) - 12100$$

≃ 1300 Psia

Scaling of temperature:

Given:

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Initial prototype reservoir temperature, $(T_r)_p = 105 \,^{\circ}F$ Initial model reservoir temperature, $(T_r)_M = 105 \,^{\circ}F$ Prototype steam temperature at 1300 Psia $(T_s)_P = \frac{577 \,^{\circ}F}{300 \,^{\circ}F}$ Model steam temperature =

Temperature ratio =
$$\frac{(\Delta T)_{p}}{(\Delta T)_{M}} = \frac{577 - 105}{300 - 105} = \frac{472}{195} = 2.42$$

Therefore for any model temperature, the prototype temperature is given by:

$$\frac{(T - T_r)_p}{(T - T_r)_M} = 2.42$$

$$T_P = 2.42 (T - T_r)_M + (T_r)_P$$

= 2.42 $T_M - 254.1 + 105$

Or

$$T_P = 2.42 T_M - 149.1....3.86$$

Where

 T_{M} = any model temperature

 $T_{p} = prototype temperature °F$

Therefore for a model temperature of 300°F the prototype temperature, $\rm T_{\rm P}$ is given by

$$T_{P} = 2.42 (300) - 149.1$$

~ 577°F

Scaling of time:

Scaling parameter IV in table 3.63 is used to convert the ptototype time to model time and vice versa, illustrated as follows:

$$\left(\frac{k_{hR} t_{R}}{\phi_{R} S_{R} \rho_{R} C_{R} L_{R}^{2}}\right)_{p} = \left(\frac{k_{hR} t_{R}}{\phi_{R} S_{R} \rho_{R} C_{R} L_{R}^{2}}\right)_{M}$$

Given:

 $k_{hp} = 1.2 \text{ Btu/hr-ft-}^{F}$ $k_{hM} = 0.5 \text{ Btu/hr-ft-}^{F}$ $(\rho_r C_r)_M = (\rho_r C_r)_p$ Substituting these values, we have

$$\frac{t_{\rm M}}{t_{\rm p}} = \left(\frac{1.2}{0.5}\right)(1)\left(\frac{1}{200}\right)^2 = 6.00 \ \text{x} \ 10^{-5}$$

or

$$\frac{t_{M}}{t_{p}} = (6.00 \times 10^{-5})(365.25 - \frac{\text{days}}{\text{years}})(\frac{24 \text{ hrs}}{\text{day}})(\frac{60 \text{ min.}}{\text{hr.}})$$

= 31.55 minutes/year

This means that 31.55 minutes in the model are equivalent to one year in the field.

Scaling of permeability:

Scaling parameters V in table 3.63 is used to convert prototype permeability to model permeability and vice versa, as illustrated below:

$$\left[\frac{\stackrel{\phi_{R}}{\underset{R}{\overset{\rho_{R}}{\overset{\rho_{R}}{\overset{\rho_{R}}{\overset{\sigma_{R}}{\overset{\sigma_{R}}{\overset{\sigma_{R}}{\overset{\tau_{R}}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{L}}}{\overset{\tau_{R}}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}{\overset{\tau_{R}}}{\overset{\tau_{R}$$

or

- - -

$$\frac{k_{M}}{k_{p}} = \frac{\dot{\phi}_{M}}{\phi_{p}} \cdot \frac{\Delta S_{M}}{\Delta S_{p}} \cdot \frac{L_{M}}{L_{p}} \cdot \frac{\mu_{M}}{\mu_{p}} \cdot \frac{\dot{\rho}_{op}}{\rho_{oM}} \cdot \frac{t_{p}}{t_{M}} \cdot \dots \cdot \dots \cdot 3.88$$

If
$$k_{\underline{M}} = 4$$
 darcies

Then for a prototype pressure of 100 Psia, substitution of previously calculated values in equation 3.88 yields:

$$k_{p} = 4 \left(\frac{.24}{.29}\right) \left(\frac{.7}{.85}\right) \left(\frac{200}{1}\right) (1) (1) (6 \times 10^{-5})$$

= .0327 darcies

Scaling of injection and production rates:

Scaling parameter VI in table 3.63 is used to convert prototype injection and production rates to model rates and vice versa, as given below:

$$\left(\frac{\mathbf{w}_{R} \mathbf{t}_{R}}{\mathbf{\rho}_{R} \mathbf{\phi}_{R} \mathbf{S}_{R} \mathbf{L}_{R}^{3}}\right)_{p} = \left(\frac{\mathbf{w}_{R} \mathbf{t}_{R}}{\mathbf{\rho}_{R} \mathbf{\phi}_{R} \mathbf{S}_{R} \mathbf{L}_{R}^{3}}\right)_{M}$$

or

$$\frac{\mathbf{w}_{M}}{\mathbf{w}_{p}} = \frac{\rho_{oM}}{\rho_{op}}, \frac{\mathbf{L}_{M}^{3}}{\mathbf{L}_{p}^{3}}, \frac{\phi_{M}}{\phi_{p}}, \frac{\Delta S_{M}}{\Delta S_{p}}, \frac{\mathbf{t}_{p}}{\mathbf{t}_{M}}, \dots, 3.89$$

Substitution of previously known values in equation 3.89, gives

$$\frac{w_{M}}{w_{p}} = (\frac{1}{200})^{3}(\frac{.29}{.24})(\frac{.85}{.70})(\frac{1}{600 \times 10^{-5}})$$
$$= 3.057 \times 10^{-3}$$

$$\frac{w_{M}}{w_{p}} = (3.057 \times 10^{-3}) (\frac{110.4 \text{ cm}^{3}/\text{min.}}{\text{bbl/day}})$$
$$= 0.336 \frac{\text{cm}^{3}/\text{min.}}{\text{bbl/day}}$$

Or

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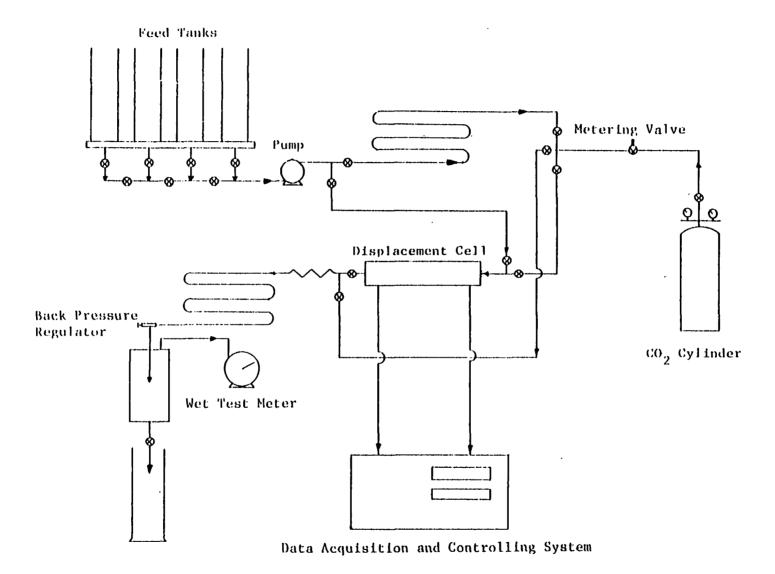
CHAPTER IV

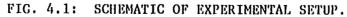
EXPERIMENTAL EQUIPMENT AND MATERIALS

An extensive experimental facility was constructed for investigating the effects of simultaneous injection of steam and CO_2 on the recovery of heavy oils. The laboratory equipment used in the displacement tests, as shown in figure 4.1, includes a linear displacement cell, feed tanks, a positive displacement metering pump, heating tapes, high pressure CO_2 cylinders along with measuring, recording and controlling devices. The important components of the facility, designed with enough flexibility to allow for varying degrees of complexity in the experimental process, are describled as follows:

1. Linear Displacement Cell:

The experiments were conducted in a stainless steel cell equipped with Hassler-type Core holders and one thermocouple at each end to measure the temperature of the flowing fluid. The cell was 24 inches in length, 3 inches in internal diameter and 0.25 inches in wall thickness with end caps screwed on the cell and sealed with high pressure, high temperature corrosion resistant 0-rings. There was one swagelock quick disconnect at each end to maintain pressure inside the cell when it was taken out from the assembly for weighing. The cell was wrapped with an





1.1

asbestos insulating tape to reduce heat losses. The system is shown in figure 4.2.

2. Porous medium:

It consisted of 20-40 mesh Halliburton frac sand. The sand pack was prepared by pouring sand in the core holder while it was constantly shaken to provide almost a tight, homogeneous and consistent reservoir. While packing sand acetone was periodically added to facilitate its settling. The sand was held in place by a metal screen followed by an 0-ring and an end cap on each side of the core holder.

3. Feed tank:

Four feed tanks made of plexi-glass served as reservoirs for oil, water, solvent and brine. All the tanks were connected to the inlet side of the positive displacement metering pump through a system of valves and the feed line as shown in fig. 4.1. 4. Positive displacement metering pump:

A positive displacement metering pump was used to inject fluids at the desired rate into the displacement cell. The outlet of the pump was connected to two lines through a T connection. One of the lines was wrapped with heating tapes and properly insulated to generate steam and was connected to the displacement cell through a system of valves and a flexible disconnect. The other line was connected directly to the displacement cell and was used to saturate the core with oil. The system is shown in figure.^{4.1}

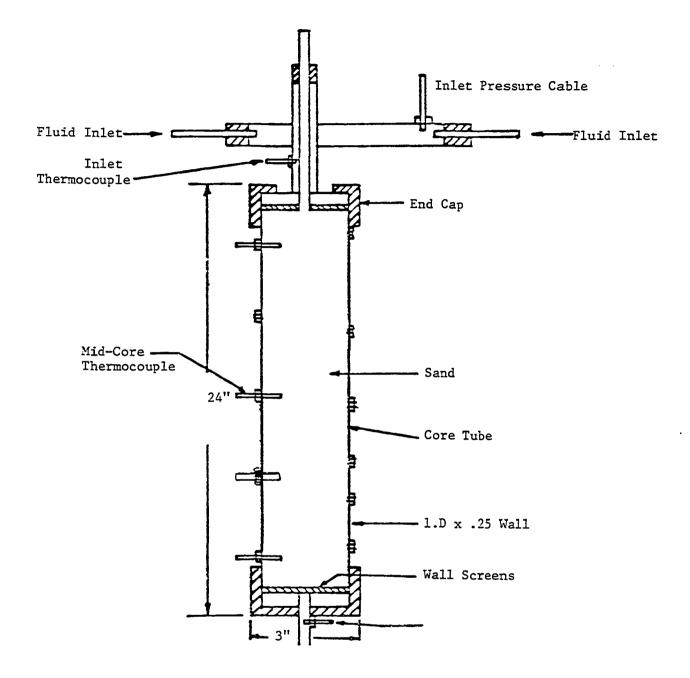


FIGURE 4.2: SCHEMATIC OF THE DISPLACEMENT CELL

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5. CO₂ cylinder:

A high pressure CO₂ cylinder, equipped with forward pressure regulator, was connected to the steam generating line through a metering valve used to control the flow rate of CO₂ to the system. 6. Pressure monitoring:

The fluid pressure in the flow system was monitored by pressure gauges and pressure transducers. The pressure transducers were connected to a validyne digital transducer indicator equipped with digital display and analog DC output proportional to the input pressure signal.

7. Temperature monitoring:

The fluid temperature in the flow system was monitored by an array of five thermocouples. Two of them were connected to two love temperature controllers through their respective heating tapes to control the temperature of the generated steam and to regulate heat to the displacement cell to set it at initial reservoir temperature. The remaining four were connected at 3. 7, 11, and 19 inches from the inlet of the displacement cell to monitor temperature at these points.

8. Production system:

The produced fluids passed through a pressure gauge into the production facilities consisting of a heat exchanger, a back pressure regulator, a sealed tubing to allow separation of CO₂ and liquids produced, a wet test meter and several valves and graduated cylinders.

The heat exchanger was built by placing about 4 ft. of the production line inside a tank full of water cooled by a

refrigeration system.

Fluids out of the heat exchanger passed through the back pressure regulator into the sealed tubing fitted with a rubber stopper at the top and a value at the bottom. The liquids collected at the bottom were produced into the graduated cylinders through the value while CO_2 escaping through a top line in the rubber stopper passed through a wet test meter and to the atmosphere.

The backpressure regulator and the wet test meter were used to control the pressure of the system and to measure the amount of CO_2 produced respectively.

CHAPTER V

EXPERIMENTAL PROCEDURE

The gravity and viscosity of the crude oil samples used in this experimental investigation were 15°, 20° and 26°API at 75°F respectively.

All the tests were carried out in a stainless steel coreholder of 24 inches in length and 3 inches in diameter equipped with 3 thermocouples, one at each end and one at the center, and packed with unconsolidated 20-40 mesh Halliburton frac. Sand of specific gravity 2.65. Total or absolute porosity of the sand was determined by using a pycnometer and the following equation.

Where

 $\phi_{abs} = Absolute porosity$ $V_{sg} = Sand grain volume$ $V_{b} = Bulk volume$

1. Before The Run:

The following steps were taken before the commencement of each run :

- i. The displacement cell was packed with sand. This was done by shaking and tapping the cell with mallet continuously, while the sand was being poured in, to provide a tight, homogeneous and consistent reservoir. After packing the excess sand was removed, the end plates of the cell were screwed inplace and the weight of the cell recorded.
- ii. The cell was connected to the flow lines after flushing them with water and then it was pressure tested for leakes. This was done by closing the inlet valve and pulling a vaccum on the cell followed by closing the outlet valve. A stable pressure in the cell indicated no leaks.
- iii. All thermocouples were connected to the recorder and checked to be in good condition.
 - iv. The inlet valve was, now, opened and water was injected at the minimum possible rate to avoid the creation of flow channels. After breakthrough water was allowed to flow for some time when the flow rate was measured and the corresponding differential pressure across the cell recorded. By using Darcy's law and this data, the absolute permeability was determined that ranged between 3 and 4.5 darcies.
 - v. The cell was disconnected from the assembly and weighed to calculate the pore volume and effective porosity as follows:

$$P.V = \frac{W_2 - W_1}{\rho_w}$$

$$\phi_{\text{eff}} = \frac{P.V}{V_{\text{b}}}$$

Where

 W_1 = Weight of cell + sand W_2 = Weight of cell + sand + water ρ_w = Density of water P.V.= Pore volume V_b = Bulk volume of sand pack

vi. The cell was mounted back and heated by turning on the thermal tapes wrapped around it and setting the temperature controller at the initial reservoir temperature of 105°F. The oil, heated by thermal tapes to reduce its viscosity and facilitate injection, was now allowed to flow through the cell to irreducible water saturation. The core was allowed to cool, disconnected from the assembly and weighed again to determine the initial oil and irreducible water saturations as follows:

$$(1-S_{wi}) \cdot \rho_0 \cdot (P.V) + S_{wi} \cdot \rho_w \cdot (P.V) = W_{firr}$$

Where

S_{wi} = Initial or irreducible water saturation

 ρ_{o} = Density of oil

 ρ_{xx} = Density of water

- vii. The injection and production lines were cleaned by flowing solvent through them and then flushing them with distilled water. The cell was connected back to the assembly and isolated from the flow lines by closing the inlet value and opening the by-pass value.
- viii. The pump and the thermal tapes on the injection lines were turned on to generate steam and the temperature controller was set at the desired steam temperature. The flow rate and pressure were adjusted to required values by regulating the stroke of the pump and the back pressure valve.
- 2. During The Run:

a. Steam Drive Only:

As the pressure, temperature and the flow rate of the generated steam stabilized at the pre-set values, the by-pass valve was closed and the steam was flowed into the displacement cell by opening the inlet valve. The temperature and pressure recordings were made by the data acquisition system. The producing line was heated when the produced fluids were

W_{firr} = Weight of fluids in the cell at the irreducible water saturation.

cold to facilitate their flow and cooled when they became hot after steam breakthrough. The produced fluids were collected in graduated cylinders and the time of production for each cylinder recorded. The run was continued until the oil/water ratio dropped to a value that could not be measured.

Injection Of Steam In Combination With CO₂: Ъ. CO_2 was mixed with steam in the injection line and its rate and pressure were controlled by the metering valve and the forward pressure regulator mounted on the CO₂ cylinder respectively to obtain a mixture of steam and CO_2 of required proportions at the required pressure. This was confirmed by measuring the volume of the condensed water collected at the production end and the volume of CO_2 produced as indicated by the wet test meter in a given time. As the pressure. temperature, and injection rates stabilized at the desired values, the bypass valve was closed and the mixture allowed to flow through the displacement cell by opening the inlet valve. The rest of the procedure was the same as described in the steam drive process except that the produced CO₂ was flowed through the wet test meter into the atmosphere as illustrated in figure 4.1.

3. After The Run:

The system was turned off and the rates and volumes of the produced fluids recorded. The produced fluids were in the form of milky brown emulsions, and some of them were very

hard to break. A little Amoco surfactant was added to facilitate separation of oil & water. The cell was unpacked, cleaned and re-packed for the next run.

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CHAPTER VI

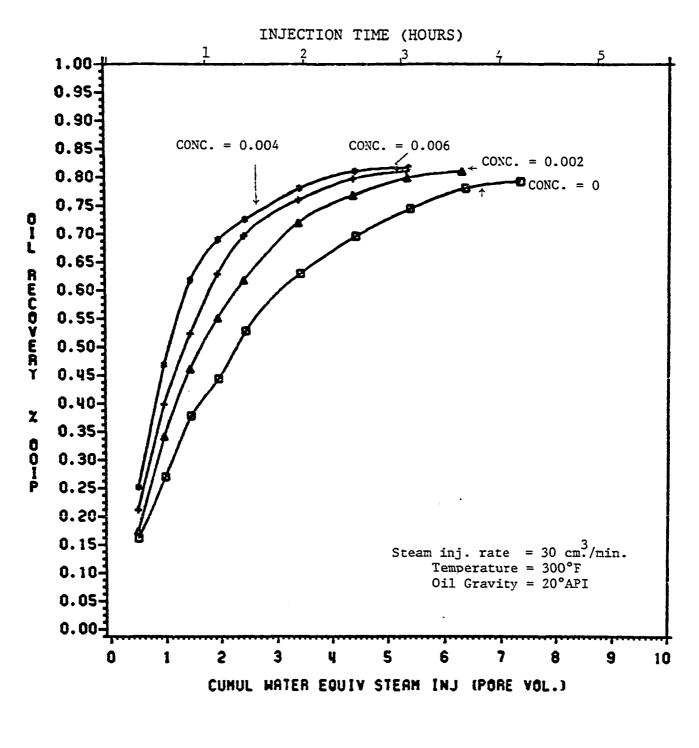
RESUTLS AND INTERPRETATIONS

The primary objective of this study was to investigate the use of CO_2 , when injected simultaneously with steam in varying proportions, as a means of improving oil recovery. With this end in view a total of 39 experiments were conducted to establish an optimum range of CO_2 /steam ratio and to determine the effects of injection pressure (temperature), rate, viscosity and pH on the oil recovery.

The experiments were carried out in five phases as follows:

In phase 1 only steam, at temperatures of 300,350,400,450, 500, & 550°F, was used to displace a 20°API oil. The data obtained from this phase is recorded in tables B-11 through B-17 Appendix B.

In Phase 2 a mixture of steam and CO_2 adjusted to a predetermined temperature and CO_2 /steam ratio was injected continuously into the core and the corresponding recoveries of the 20°API oil were recorded. The results of the runs conducted in this phase are given in tables B-21 through B-47. The data of tables B-11 through B-17 and B-21 through B-47 was integrated together and plotted as shown in figures 6.1 and Gl through G5 appendix G to evaluate the effects of CO_2 , when injected along with steam, on the oil recovery.



:

FIGURE 6.1: EFFECT OF CO_2 ON STEAM DRIVE RECOVERY CONC. = CO_2 /STEAM RATIO (SCF CO_2 /CM³ STEAM INJ.)

In phase 3 the injection rate of steam, at a fixed temperature of 400°F and optimal CO_2 /steam ratio, was varied and the corresponding recoveries of the 20°API oil were recorded in tables C-ll through C-l4. The data obtained so far was used to determine the effect of injection rate on oil recovery as illustrated in figure 6.2.

In phase 4, 15° and 26°API oils were used and the tests were run as in phases 1 and 2. The results obtained from this series of experiments are given in tables D-11 through D-19 and plotted as shown in figures 6.3 and G6 through G7 appendix G to evaluate the effect of viscosity of the oil, at the steam temperature, on oil recovery.

In phase 5 alkaline water of pH 12 was used for the steam generation and the experiments were conducted similar to the ones described in phases 1 and 2. The results obtained are given in tables E-11 through E-15 and plotted in figures G8 through G11 appendix G to evaluate the effect of high pH on oil recovery from the core when subjected to simultaneous injection of CO₂ and steam. 6.1 PHASE 1: Determination of recovery by conventional steam

flooding.

The results obtained from this series of experiments, as summarized in tables B-ll through B-l7 and plotted in figures 6.1 and Gl through G5 appendix G are used as a criteria of comparison with the results of all other experiments conducted in the subsequent phases to find out if the addition of CO₂ in steam improves recovery. A total recovery of 79.2% of the original oil in place was obtained when steam was injected at 300°F. The total recoveries at injection temperatures of 350, 400, 450, and 550°F were 76.8, 73.0, 67.8, 65.3, and 63.0% respectively. The purpose of making the runs at different temperatures was to evaluate the effect of pressure and temperature on oil recoveries which will be discussed later in this chapter.

6.2 PHASE 2: Determination of CO₂ concentration in the injected steam to maximize recovery.

Results from this study are summarized in tables B-21 through B-47 and plotted in figures 6.1 and Gl through G5 appendix G. The examination of these figures indicates that at a CO_2 /steam ratio of .004 SCF GO_2 /cm³ steam, the oil recovery is maximized at all temperatures. The total recovery at 300°F is 81.6% which is only 2.4% higher over the conventional steam flooding process. However, it may be noted that the injection of CO_2 increases the rate of recovery significantly as illustrated in figure 6.1. An overall recovery of 79.2% is obtained in about 4.2 hours with conventional steam flooding, while only about 2.15 hours are needed to realize the same amount of recovery when a mixture of CO_2 and steam (.004 SCF CO_2 /cm³ steam) is injected.

The additional recovery obtained by carbon dioxide injection with steam is attributed to the:

 High solubility of carbon dioxide in oil which results in swelling of the oil ahead of the steam front thus decreasing its viscosity and increasing its permeability, both favoring a more efficient displacement.

- Solubility of carbon dioxide in water increases its viscosity and improves water-oil mobility ratio.
- 3. Improved miscibility at the condensation front due to enhanced steam distillation behind the heat front.
- 4. Lowering of interfacial tension and promotion of miscibility effects that promotes pistonlike displacement.
- Better heat distribution and additional volumetric sweep provided by the injected gas.
- 6. Trapped gas effect.
- 7. Solution gas drive effects.

Generally carbon dioxide is not miscible on first contact with reservoir oils, but may develop miscibility through multiple contacts at sufficiently high pressure depending upon the reservoir temperature and oil characteristics. Holm and Josendal reported that miscibility may be achieved in case of light oils at pressures of the order of 2000-3000 psig, but with very viscous oils the miscibility pressure can never be reached as shown in figures Gl2 and Gl3 appendix G. The examination of these figures also indicates the dependence of minimum miscibility pressure on temperature. The miscibility pressure requirement increases with increasing temperature.

This experimental study was conducted at such pressures and temperatures that carbon dioxide miscibility requirements with oil are not met. However, even without miscibility the swelling caused by the dissolution of carbon dioxide in the oil decreases its viscosity and increases its permeability, both of which favor a more efficient displacement.

As soon as the mixture of steam and carbon dioxide enters the experimental cell, it starts rapidly migrating upward due to strong gravitational gradients while advancing into the originally cool reservoir. The injected steam heats the formation and a fraction of crude oil in the steam zone vaporizes. The vaporization process is further enhanced by the presence of carbon dioxide in the injected steam. The hydrocarbon vapor is carried forward through the advancing mixture of steam and carbon dioxide. The steam and the hydrocarbon vapor condense and mix with the original crude at the condensation front to form a hot water zone and a hydrocarbon distillate or solvent bank; whereas the noncondensable carbon dioxide, some of which gets dissolved in the water and oil phases, creates a permanent gas phase along the top of the reservoir. All these processes assist in enhancing and improving recovery.

The distillate bank drives the oil miscibly ahead of the steam front and the solubility of carbon dioxide in water increases its viscosity and thus improves water-oil mobility ratio. The dissolved carbon dioxide in oil breaks out of solution with decreasing pressure, as the front moves towards the producing end, and consequently provides additional drive energy. The overlying permanent gas phase provides additional sweep and assists in propagation of steam thus resulting in earlier arrival of heat at the producing end. This heats up the formation close to the producing end much sooner than the conventional steam flooding process thus resulting in accelerated oil production due to increased mobilization of oil caused by viscosity

reduction. A schematic diagram of the CO₂/steam injection system is shown in figure Gl4 appendix G.

The examination of figures 6.1 and Gl through G5 appendix G indicates that oil recovery is also affected by the level of concentration of carbon dioxide in the injected steam. As the carbon dioxide concentration increases from .002 to .004 SCF CO_2/cm^3 . steam the recovery also increases, but a further increase in concentration of carbon dioxide from .004 to .006 SCF CO_2/cm^3 steam results in a decreased recovery. The first increase in concentration from .002 to .004 SCF/cm³ steam, increases the amount of carbon dioxide dissolved in the oil phase, but when the concentration is increased from .004 to .906 SCF/cm³ steam, very little additional carbon dioxide goes into solution. The dissolved gas helps to decrease the viscosity of oil and provides additional energy to move it towards the producing end. Whereas the undissolved gas simply builds up the gas saturation resulting in increased relative permeability to the gas. With increasing concentration of carbon dioxide, the decreasing oil viscosity contributes towards the improvement in recovery while the increasing relative permeability to gas lowers the ultimate recovery. The counteracting effects of both these parameters result in an optimum concentration of about .004 SCF CO_2/cm^3 steam observed in this study. 6.3 PHASE 3: Determination of the steam injection rate required

to maximize recovery.

The results from this study are summarized in tables C-ll through C-l4 and plotted in figure 6.2. The examination of this figure indicates that the recovery is maximized at the steam injection rate of 30 cc/min. The probable explanation for the poor

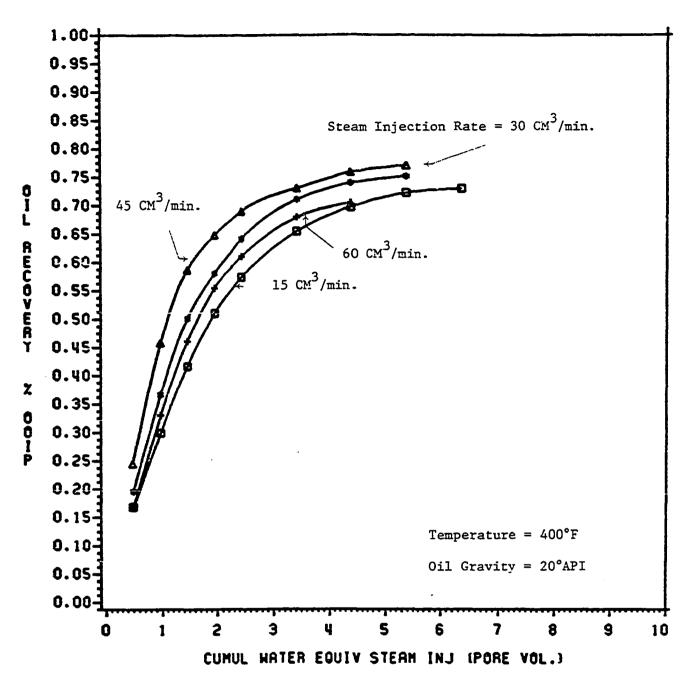


FIGURE 6.2: EFFECT OF INJECTION RATE ON STEAM DRIVE RECOVERY $CO_2/STEAM$ RATIO = .004 SCF CO_2/CM^3 STEAM INJECTION

efficiency exhibited at very low and high rates is as follows:

A very low rate of steam injection implies a very low rate of heat injection into the reservoir which causes the steam zone to develop very slowly thus lowering the possibility of developing a steam drive. In other words the process will be equivalent to a hot water drive with all its accompanying inefficiencies in terms of oil/water ratio.

With continuing increase in the steam injection rate, the steam zone grows accordingly until a steam drive is developed accompanied by its unique efficiency in terms of oil/water ratio. At high injection rates, the steam zone which increases relative permeability to highly mobile steam vapor and promotes steam channeling to the production end, thus lowering the recovery efficiency significantly.

6.4 PHASE 4: Effect of oil Gravity.

Figure G15 appendix G indicates that at any given temperature, the viscosity of an oil is a function of its gravity. A low API gravity oil is more viscous than a higher API gravity oil under similar conditions. Also the examination of figure G16 shows the effect of viscosity on the rate of development and the degree of override. As the steam enters the formation, it starts miggating upwards because of the marked difference in density between the steam and the reservoir fluids. The rate of this upward migration is dependent on oil viscosity at steam temperature in addition to vertical permeability. The higher viscosity

dictates higher rates of upward migration and more pronounced gravity override effects which in turn implies a poor displacement efficiency and poor recovery.

The above discussion points ou that the viscosity of oil and hence the API gravity are very important parameters affecting the recovery efficiency that increases with decreasing viscosity.

Carbon dioxide dissolves in crude oil and when it goes into solution, the volume of the oil increases and its viscosity decreases significantly. The amount of swelling and the reduction in viscosity depends on the crude oil gravity as shown in figures G15 and G17 appendix G. Generally speaking, the lower the API gravity of the oil the greater the percentage reduction that takes place in the viscosity on dissolution of cargon dioxide in the oil. Thus, viscosity reduction is significant and more pronounced with medium and heavy oils than with the light oils which leads to higher incremental recovery due to simultaneous injection of carbon dioxide and steam over the conventional steam flooding process under similar conditions of temperature and pressure.

The effect of oil gravity was examined by displacing three oils of gravities 15°, 20°, 26° API with a mixture of carbon dioxide and steam at 400°F. The results thus obtained are given in tables D-11 through D-19 and plotted in figures 6.3 and

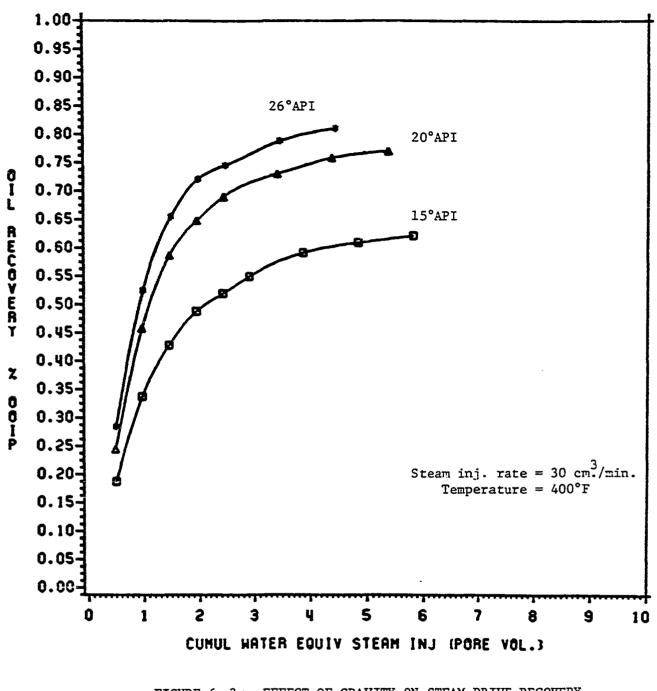


FIGURE 6.3: EFFECT OF GRAVITY ON STEAM DRIVE RECOVERY $CO_2/STEAM RATIO = .004 SCF CO_2/CM^3 STEAM INJ.$

G6 and G7 appendix G. The examination of these figures indicates that the optimum CO_2 /steam ratio of .004 SCF CO_2 /cm³ steam still prevails, but the total recovery increases with decreasing oil gravity. The total recovery of 15°API oil at the optimum CO_2 /steam ratio is about 8% higher than the case when no carbon dioxide is injected with steam. In case of 20°API oil the improvement in recovery is only 4% while there is no improvement in ultimate recovery in case of 26°API oil. This is, because, the percentage reduction in viscosity due to dissolution of carbon dioxide in oil and even distribution of heat decreases with increasing API gravity, thus giving rise to lower incremental recoveries in case of higher API gravity oils.

6.5 PHASE 5: Effect of pH.

The effect of pH on oil recovery was examined by displacing a 20° API oil first with caustic steam (pH = 12) alone and then with varying levels of carbon dioxide concentrations in the injected steam at 400°F. The results thus obtained are given in tables E-12 through E-15. The results of similar experiments when conventional steam (pH = 8.5) was used as a displacing fluid are given in tables B-14, B-24, B-34, and B-44 respectively. The data of tables E-12 and B-14 is plotted in figure G8 to evaluate the effect of pH on oil recovery when no carbon dioxide is injected with steam. Similarly, data of tables E-13/ B-24, E-14/B-34, E-15/B-44 is plotted in figures G9 through G11 respectively to determine the effect of pH on oil recovery

when steam is injected with varying concentrations of carbon dioxide. The examination of these figures indicates that:

- Although there is no significant improvement in ultimate recovery, the rate of recovery increases significantly with increasing pH when no carbon dioxide is injected with steam (fig. G8).
- Neither the rate of recovery nor the ultimate recovery is affected by pH when carbon dioxide is injected in combination with steam (fig. G9 through Gll).

The process of caustic flooding has been studied by various investigators who reported that the incremental recovery of caustic flooding over conventional steam flooding results from:

- 1. Lowering of interfacial tension
- 2. Reversal of rock wettability
- 3. Emulsification and entrapment

The results of this study are in agreement with the findings of these investigators when steam is injected alone to recover oil. The probable reasons that the recovery is not affected by pH when carbon dioxide and steam are injected together, are as follows:

 Carbon dioxide dissolves in water to form carbonic acid according to the following equilibrium relationship

 $CO_2 + H_2 O \longrightarrow H_2 CO_3$

which reduces pH.

2. The displacement mechanisms of carbon dioxide dominate the displacement mechanisms of the caustic flooding.

Although it is not part of this study, the importance of steam injection process to recover heavy oil makes it necessary to understand the effect of high pH and high temperature on the reservoir minerals and fluids. Laboratory studies have indicated that the injection of high pH fluids cause substantial dissolution of reservoir minerals. These minerals are carried forward through the advancing fluids and reprecipitate in the pore spaces as the temperature and pH falls, thus causing significant reduction in permeability of the areas away from the injection end. Incompatibility of the injected and formation water causes chemical reactions between the dissolved salts resulting in the production of precipitates which can reduce permeability too. Generally, clays are more compatible with low pH fluids than with high pH fluids, which cause expansion and dispersion of water-sensitive clays and thus reduce the formation permeability the production of solid particles in the produced fluids, plugging of surface equipment, and deterioration of gravel packs and liners are some of the other problems associated with the injection of high pH fluids. To minimize this kind of damage it is advisable to maintain the pH of the injected fluids as low as possible. The formation of carbonic acid due to carbon dioxide injection may jelp to reduce pH and moreover because of the neutratization the corrosion problems associated with carbon dioxide injection may be reduced.

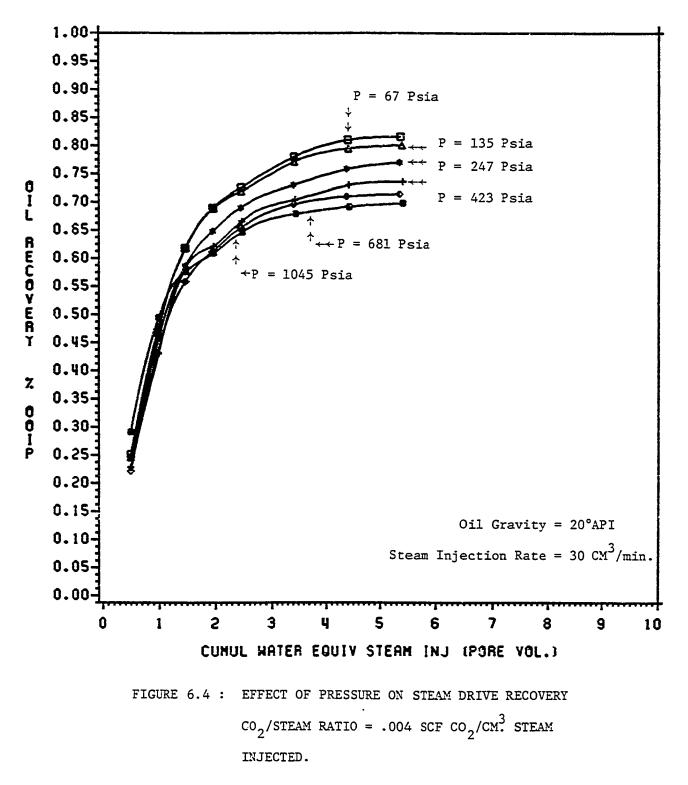
6.6 EFFECT OF PRESSURE:

The effect of pressure on oil recovery is examined by studying figure 6.4 plotted by using data obtained in phase 2. These results indicate that the total recovery decreases with increasing pressure. A possible explanation for this is as follows.

Pressure and temperature are inter related in steam flooding. The solubility of carbon dioxide in crude oil is affected both by temperature and pressure. It increases with increasing pressure while decreases with increasing temperature, both parameters counteracting the effects of each other. For a given quality of steam, a high pressure is coupled with high temperature with resulting low viscosity of oil and greater driving force for displacement, both mechanisms contributing towards improvement in recovery. But at high pressure, the resulting lower specific volume of steam is coupled with lower flow rate and hence a lower recovery.

The latter effect dominates the former with the result that the total recovery is reduced at high pressures. But in case of very viscous oils, a greater driving force for displacement is necessary to ensure the desired oil mobility which can only be developed by high pressure steam injection. Therefore, it is recommended that all steam flood operations should be conducted at the lowest possible pressure to insure the efficient use of steam vapor.

EFFECT OF PRESSURE ON STEAM DRIVE RECOVERY



6.7 COMPARISON WITH THEORY:

In figure 6.5, the oil recoveries calculated by using the Myhill and Stegemeier⁴⁷ prediction model are compared with those obtained experimentally for run 3 and given in table B-14 Appendix B. Sample calculations and results for the run are presented later in this section illustrating the use of the prediction model. Generally good agreement exists between the observed and calculated values. The difference however, is insignificant and is probably due to the assumptions made in the derivation of the prediction model. Also, qualitatively similar reservoir simulation results were obtained by K.C. Hong and J.W. Ault⁶⁶. A quantitative comparison can not be made because of the different reservoir and fluid properties. However, both studies appear to follow the same general trend, thus supporting the results of this study.

In figures 6.6 and G18 through G22 the temperature distributions calculated by using the Lauwerier²² model are compared with those obtained experimentally for runs 1 through 6 and given in tables F-1 and F-2, Appendix F. Since the Lauwerier model describes the temperature distribution in a linear system with hot water injection, the deviations between the observed and calculated values are as expected indicating the development of a steam zone which makes the thermal profile steeper. A knowledge of the temperature profile helps in predicting the location and rate of advance of the steam zone. A computer programme and the computed results, based on the Lauwerier model, are given in Appendix F for runs 1 through 6.

SAMPLE CALCULATIONS USING MYHILL AND STEGEMEIER⁴⁷ PREDICTION MODEL FOR RUN 3.

DATA:

. . .

$$T_{s} = 400^{\circ}F; T_{r} = 105^{\circ}F$$

$$Pv = 1035 cc = .0365 ft^{3}$$

$$S_{oi} = 0.805$$

$$S_{wi} = 0.195 - ...$$

$$S_{or} = 0.208$$

$$L_{v} = 825.9 Btu/lb_{m} (steam tables)$$
Oil gravity = 20 °API

Heat capacity of oil

 $C_{o} = (0.388 + 0.00045 T) / \gamma_{o}$

$$Y_{0} = \frac{141.5}{131.5 + API}$$

$$= \frac{141.5}{131.5 + 20} = 0.934$$

$$C_{0} = (0.388 + .00045 + \frac{105 + 400}{2})/0.934$$

$$= 0.537 \text{ Btu/lb}_{m} - ^{\circ}\text{F}$$

$$C_{w} = \frac{h_{w} (400) - h_{w}(105)}{400 - 105}$$

$$= \frac{375.1 - 72.991}{295} = 1.0241 \text{ Btu/lb}_{m} - ^{\circ}\text{F}$$

$$\rho_{o}(400) = 62.4 (0.81) = 50.54 \text{ lb}_{m}/\text{ft}^{3}$$

$$\rho_{w}(400) = \frac{1}{.01864} = 53.648 \text{ lb}_{m}/\text{ft}^{3}$$

$$M_{w} = C_{w} \rho_{w}$$

$$= (1.0241) (53.648) = 54.941 \text{ Btu/ft}^{3} - ^{\circ}\text{F}$$

$$M_{o} = C_{o} \rho_{o}$$

$$= 0.537 * 50.54 = 27.14 \text{ Btu/ft}^{3} - ^{\circ}\text{F}$$

$$M_{g} = (\frac{\lambda}{\alpha}) = 56.13 \text{ Btu/ft}^{3} - ^{\circ}\text{F}$$

$$M_{g} = (1 - 0) M_{g} + (S_{o})(M_{o}) + \phi S_{w} M_{w}$$

$$+ S_{g} [f M_{g} + (1 - f) (\frac{\rho_{s} L_{v}}{\Delta T} + \rho_{s} C_{w})]$$

$$= (1 - .3723) 42.3 + .3723 (.208) (27.14)$$

$$+ (.3723) (.195) (54.941) + .3723 (.597).$$

$$[\frac{.5368 * 825.9}{295} + .5368 * 1.0241]$$

$$= 33.098 \text{ Btu/ft}^{3} - ^{\circ}\text{F}$$

• • • •

$$f_{hv} = [1 + \frac{C_{w \Delta T}}{f_{sdh} L_{vdh}}]^{-1}$$

$$= [1 + \frac{1.0241 (295)}{(1.) (825.9)}]^{-1}$$

= 0.7322

$$e^{t_{cd}} e^{t_{cd}} = 1 - .7322$$

= 0.2678
 $t_{cd} = 3.672$

$$t = \frac{1}{4} \left(\frac{M_R}{M_s} \right)^2 \left(\frac{\pi r^2}{\sigma_s} \right) t_D$$

= .001033 t_Ddays

.

or

.

- ---

t =
$$1.487 t_{D}$$
 minutes
 $w_{f} = 30 \frac{cm^{3}}{min.}$ or $95.2 \ lb_{m}/day$
 $\dot{Q}_{i} = 95.2 \ [1.0241 \ (295) + 825.9]$
= $107386 \ Btu/D$

$$Q_{i} = 107386 * t$$

$$v_s = \frac{Q_i \cdot E_{h,s}}{M_R \cdot T}$$

$$= \frac{107386}{33.098} \frac{Btu}{ft^3 - F} \cdot \frac{D}{24 \text{ hr.}} \cdot \frac{1 \text{ hr.}}{60 \text{ min.}} t \text{ (minutes)}$$

$$= .007638 \text{ E}_{h,s} t$$

$$= .007638 \text{ E}_{h,s} t$$

$$N_p = (S_{oi} - S_{or}) \text{ E}_c \text{ V}_s$$

$$= 0.3723 (.805 - .208) .7 \text{ V}_s$$

$$= 0.155584 \text{ V}_s \text{ ft}^3$$

or

• - -

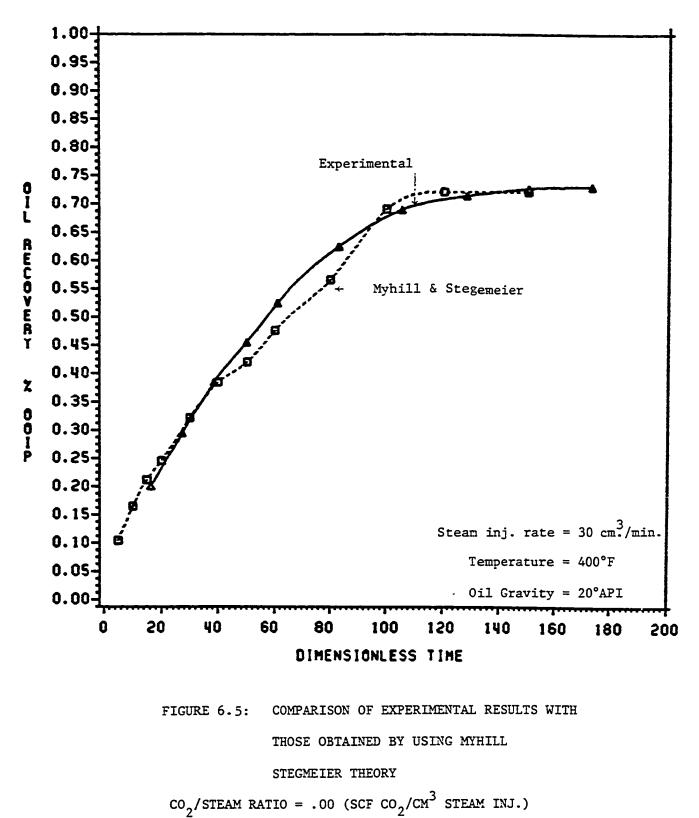
.

 $N_{p} = 4405.6 V_{s} cm^{3}$

•

t _D	t=1.487t _D	[*] Eh,s	Q _i Btu	V_{s} ft ³	N _p cc	R
5	7.435	.346	554.4	.1965	86.6	.104
10	14.87	.275	1109.0	.03123	137.6	.165
15	22.305	.235	1663.4	.04004	176.4	.212
20	29.74	.205	2217.8	.04656	205.1	.246
30	44.61	.179	3326.7	.06099	268.7	.322
40	59.48	.160	4435.6	.07269	320.2	.384
50	74.35	.14	5544.5	.07951	350.3	.420
60	89.22	.132	6653.4	.08995	396.3	.476
80	118.96	.118	8871.3	.10722	472.4	.566
100	148.7	.115	11089.1	.13062	575.5	.691
120	178.44	.1	13306.9	.13629	600.4	.721
150	223.05	.08	16633.6	.13629	600.4	.721

^{*}Values of E_{h,s} are obtained from figure G23.



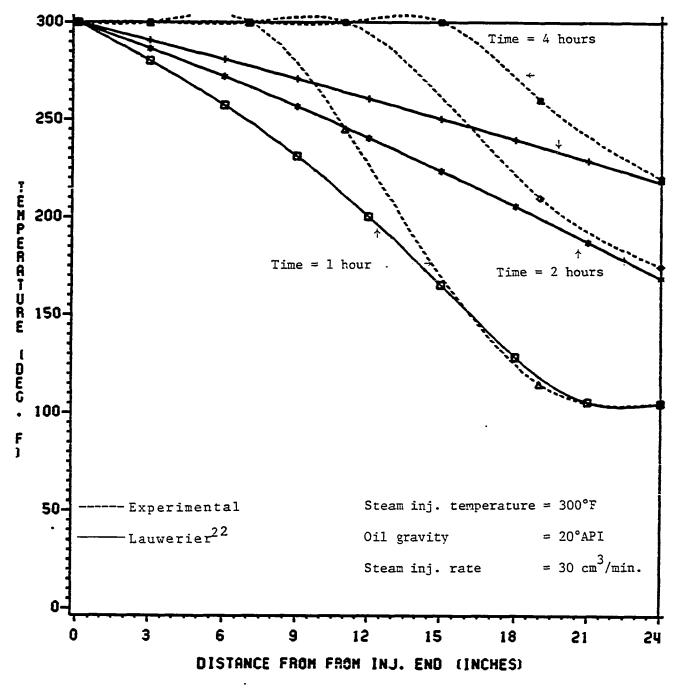


FIGURE 6.6: TEMPERATURE DISTRIBUTION WITH DISTANCE FOR RUN NO. 1

CHAPTER VII

SUMMARY AND CONCLUSIONS

Summary and Field Applications:

The injection of carbon dioxide with steam results in significantly increased production rates. This is due to (a) additional sweep and oil viscosity reduction and (b) better heat distribution provided by the injected gas. Simultaneous injection of carbon dioxide and steam also results in the improvement of ultimate recovery of heavy oil which may be as high as eight percent over the conventional steam flooding process. This increase in the ultimate recovery is attributed to:

- a. The swelling of the crude oil with the dissolved carbon dioxide ahead of the heat front, thus resulting in decreased oil viscosity and increased oil permeability, both favoring a more efficient displacement of the oil.
- b. Improved oil mobilization due to better heat distribution and additional volumetric sweep provided by the injected gas.
- c. Enhanced steam distillation behind the heat front.
- d. Reduced interfacial tension.

- e. Trapped gas effect.
- f. Solution gas drive.

The results show that recovery is affected by carbon dioxide concentration in the injected steam. Increasing its concentration from .002 to .004 SCF CO_2/cc steam, increases the amount of CO2 dissolved in oil, but when the concentration is increased from .004 to .006 SCF CO_2/cc steam, very little additional CO2 goes into solution. The dissolved gas helps to decrease the viscosity of oil and provides additional energy to move it towards the producing end, while the undissolved gas simply builds up the gas saturation resulting in increased relative permeability to gas. The decreasing oil viscosity contributes towards improvement in recovery and the increasing relative permeability to gas lowers the ultimate recovery. The counteracting effects of both these parameters results in an optimum concentration of CO₂ which is about .004 SCF CO_2/cc steam observed in this study. The results obtained show the importance of the steam injection rate. At very low injection rates, the recovery is low due to very low rates of heat injection into the reservoir, which causes the steam zone to develope very slowly thus lowering the possibility of a steam drive. At very high injection rates the recovery is low due to very rapid growth of steam zone which increases the relative permeability to highly mobile steam. The study suggests the existence of an optimal steam injection rate at which the recovery is maximized. This was found to be around 30 cc/min.

The experimental results indicated the importance of the viscosity and gravity of the crude oils at steam temperature in affecting ultimate recovery. Generally, a low gravity oil is more viscous than a high gravity oil. It was observed that the ultimate recovery decreased with increasing viscosity of the crude. This was because of the displacement efficiency, which decreases with increasing viscosity due to more pronounced gravity override. But the incremental recovery due to simultaneous injection of CO_2 and steam over the conventional steam flooding process, under similar conditions of temperature and pressure, was higher for more viscous oils because of the fact that the percentage reduction in viscosity resulting from the carboration of these oils is greater and more pronounced than for the less viscous oils.

A change in pH has no effect on recovery when CO₂ is added to the injected steam. However, when steam is injected alone the rate of recovery is slightly increased with increasing pH, but there is no significant improvement in ultimate recovery.

The ultimate recovery decreases with increasing pressure and hence the temperature for a given quality of steam due to the competing effects of viscosity reduction and reduced volumetric sweep, in which the latter effects dominate the former. The solubility of CO₂ in crude oil increases with increasing pressure but decreases with increasing temperature, both parameters counteracting the effects of each other.

For a given quality of steam a high pressure is coupled with high temperature with resulting low viscosity of oil and a greater driving force for displacement, both mechanisms contributing towards improvement in recovery. But at high pressures the volumetric sweep is reduced due to lower specific volume of steam coupled with lower flow rate thus resulting in lower recovery. The latter effect dominates the former and hence the total recovery is reduced at high pressures.

In case of very viscous oils, a greater driving force for displacement is necessary to ensure the desired oil mobility which can only be developed by high pressure steam injection. Therefore it is recommended that all steam flood operations should be conducted at the lowest possible pressure to ensure the efficient use of steam vapor. In this connection it is suggested to start a steam flood at a high pressure necessary for the mobilization of oil and should be gradually reduced to the full realization of the process. The requirement for high pressure CO_2 /steam injection can be greatly reduced if it is preceded by cyclic steam stimulation process which reduces resistance to flow near the producing end and thus facilitates communication between the injection and producing end.

It should be pointed out that recovery is affected by the boundary of the reservoir. These effects can be minimized by using longer laboratory cells if space limitations allow. Closmann et al⁵⁵ reported a higher recovery factor for longer

sand packs than for shorter ones. This is due to the lengths of the mixing zones which are a much longer fraction of the model lengths in case of shorter sand packs. This suggests that the results obtained in this study are pessimistic and therefore a higher recovery in the field may be expected.

The reservoir characteristics suitable for the simultaneous injection of CO₂ and steam should be such that maximum contact between the injected fluids and the resident oil could be established in order to maximize the effects of viscosity reduction, oil swelling and miscible displacement. Therefore, reservoirs with fractures, thin pay zones under-lain by large aquifers or reservoirs with large free gas caps will not be suitable for this process.

Economically it is not feasible to use downhole steam generators in very shallow heavy oil reservoirs. Therfore, the cost of external injection of CO_2 must be justified by the additional or accelerated oil recovery. But where downhole steam generators are used, the CO_2 available as flue gas can be utilized and injected along with steam.

Reservoirs containing highly asphaltic crudes and having a low permeability are not suitable for simultaneous injection of CO_2 and steam, since permeability may be damaged due to the deposition of asphaltenes resulting from the contact of crude oil by CO_2 .

Conclusions:

The following conclusions were made on the present experimental investigation.

- The injection of carbon dioxide with steam increases the
 rate of recovery significantly.
- 2. The recovery is affected by the concentration of CO₂ in the injected steam and is maximized at a concentration of about .004 standard cubic feet of CO₂ per cubic centimeter of cold water equivalent steam.
- 3. The overall recovery is dependent on oil viscosity and hence the API gravity. It improves by 8% in case of 15°API oil, 4% in case of 20°API oil, whereas no significant improvement in ultimate recovery, over the conventional steamflooding process, was observed in case of 26°API oil.
- The recovery decreases with increasing pressure and hence the temperature.
- 5. The recovery is rate dependent and is maximized at a steam injection rate of 30cm³/minute.
- 6. The recovery is not affected by pH, when steam and CO₂ are injected simultaneously. However, when steam is injected alone the rate of recovery is slightly increased with increasing pH, but there is no significant improvement in ultimate recovery.

Recommendations for further research:

It is recommended that further work in this area should be directed to find:

- The effect of cyclic steam injection prior to CO₂/steam injection process on oil recovery.
- The effect of gradual reduction of pressure after steam breakthrough on oil recovery.
- The effect of gradual reduction of steam injection rate after steam breakthrough on oil recovery.
- 4. The effect of injection of CO₂ in an alternate fashion with steam.
- 5. The effect of any other noncondensable inert gas such as nitrogen and the result compared with this study.
- The effect of pH on oil recovery when heavy oils of varying acidity are displaced by a mixture of CO₂ and steam.

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NOMENCLATURE

Symbol	Description U	Inits
A	Area	.ft ²
С	Compressibility	.psi ⁻¹
С	Concentration	$.1b_m/ft^3$
c _{co2}	Carbon dioxide concentration	$\frac{1b_{m}}{1b_{m}}$ mole
С	Isobaric specific heat	.Btu/lb _m -°F
c _o	Isobaric specific heat of oil	.Btu/lb _m -°F
С _w	Isobaric specific heat of water	.Btu/1b _m -°F
Cv	Heat capacity at constant volume	.Btu/1b _m -°F
c _σ	Isobaric specific heat of solids in reservoir matrix	.Btu/lb _m -°F
d	Grain diameter	. CM
D	Dispersion co-efficient	.ft ² /D
DM	Molecular diffusivity	.ft ² /D
e	Internal energy per unit mass	.Btu/lb _m
erfc (x)	Complementary error function	
Ec	Fraction of oil displaced that is produced	.dimensionless
Eh	Heat efficiency, the fraction of the injec- ted heat present in the reservoir	

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E _{h,s}	Steam zone heat efficiencydimensionless
fhv	Fraction of heat injected in vapor formdimentionless
fs	Steam qualitydimensionless
g	Acceleration due to gravityft/sec. ²
h	Enthalpy per unit massBtu/lb_m
h	Reservoir thicknessft.
h _n	Net reservoir thicknessft.
h _t	Gross reservoir thicknessft.
i	Injection ratebbl/D
k	Permeabilitymd
k _h	Thermal conductivityBtu/ft-D-°F
k _r	Relative permeabilitydimensionless
^k oj	Equilibrium ratios Ib _m mole b _m mole
L	Length of distanceft.
L _v	Latent heat per unit massBtu/lb_m
m	Arbitrary variabledimensionless
m	Masslb _m
М	Molecular weightlb_/lb-mole_m/lb-mole
Μ	Volumetric isobaric heat capacity (C)Btu/ft ³ -°F
M _R	Volumetric heat capacity of reservoirBtu/ft ³ -°F
Ms	2
5	Volumetric heat capacity of steamBtu/ft ³ -°F

n Number of components

x _n	Distance, direction normal to boundaryft.
Np	Cumulative oil production bbl
^N Re	Reynolds numberdimensionless
Ρ	Pressure Psi
<u>- 9p</u> - 9n	Pressure gradient normal to displacement Psi front ft.
P _{c,og}	Capillary pressure between oil and gas phasePsi
Pc,wo	Capillary pressure between water and oilPsi phase
q	Production rate, flow ratebb1/D
q	Conductive heat flux
Q	Amount of heat in reservoirBtu
Q r	Rate of energy input from sourcesBtu/D
R	Recovery%00IP
^R s	Solution gas oil ratio ^M scf/bbl
S	Saturationdimensionless
t	timeD
^t cD	Dimensionless critical time
Т	Temperature°F
u	Volumetric flux
v	Fluid velocityft/D
v	Specific volume <u>ft³</u> lb _m

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V	Volumeft ³
^V s	Steam zone volumeft ³
W	Mass rate of flowlb_m/D
x,y	Co-ordinate distancesft.
Z	Co-ordinate distance normal to bedding plane.ft.
^Z t	Reservoir thickness normal to bedding plane. ft.
x	Thermal diffusivity ft ² /D
(11)	Function equal to ratio of values of its argument (m) in the prototype to that in the model dimensionless
γ	Specific gravitydimensionless
∆()	Increment or decrement in ()
μ	ViscosityCP
ρ	Densitylb _m /ft ³
φ	Porosity
SUBSCE	RIPTS
а	Aqueous phase; apparent or effective
Ъ	Bulk; bottom hole
с	Cap or base rock; Critical; Capillary (with pressure)
С	Component
co ₂	Carbon dioxide
D	Dimensionless
е	Energy
eq	Equivalent
f	Fluid

g Gas, gas phase, gaseous phase

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gr	Grain
i	Initial; Injection
inj	Injection
j	index (j=1,2,3,); phase (oil, water or gas)
1Ъ	Lateral boundary
m	Movable saturation
М	Model
n	Normal to boundary
0	Oil
ors	Residual oil saturation
Р	Produced
р	prototype; phase
r	Reservoir rock; relative (with k)
R	Reference quantity used to obtain dimensionless number
S	Steam
Sat	Saturation temperature or pressure
Sc	Standard conditions
t	Total interval
ub	Upper or lower boundary
v	Vaporization (with latent heat)
w	Water
Wc	Connate water (irreducible water)

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APPENDIX A

PROPERTIES OF OIL AND POROUS MEDIA

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TABLE A-1

CRUDE OIL PHYSICAL PROPERTIES

Oil Gravity at 70°F	Temperature	Viscosity	
(°API)	<u>(°F)</u>	(CP)	
15	70	1950	
	150	100	
	200	41	
			
20	70	170	
	150	22	
	200	11	
26	70	4.2	
20	70	43	
	150	10	
	200	5.4	

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TABLE A2

MINERAL CONTENT BY PERCENT WEIGHT OF THE POROUS MEDIA (HALLIBURTON 20-40 FRAC SAND)

MINERAL	WEIGHT PERCENT
QUARTZ	98.5
FELDSPAR	1.0
CLAYS	0.5
Fe, Mg, Al & Ti	0.2

APPENDIX B1

SUMMARY OF DISPLACEMENT TEST RESULTS

(PHASE 1)

LEGEND FOR TABLES B12 THROUGH B47

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Column

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1	= Volume of steam injected (water eq.)	cc.
2	= Oil produced	cc.
3	<pre>= Cumulative volume of steam injected (cum. water eq.)</pre>	cc.
4	= Cumulative oil produced	cc.
5	= Cumulative oil produced % of 00	DIP.

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TABLE B11

PROPERTIES OF THE SANDPACK AND FLUIDS

Oil gravity	=	20°API
CO ₂ /Steam ratio	=	0
Steam injection rate	=	30 cc./min.

RUN NO.	P.V.	N	so _i	T s °F
	cc.	cc.		°F
1	1021	831	.813	300
2	1034	833.4	.806	350
3	1035	833.2	.805	400
4	1019	818.3	.803	450
5	1027	803.0	.782	500
6	1035	833.4	.805	550

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TABLE B-12:	DISPLACEMENT	TEST RESULTS	(RUN 1)

1	2		3		5
<u></u>	cc	cc	PV		
500	135	500	0.49	135	0.162
500	90	1000 .	0.98	225	0.27
500	90	1500	1.47	315	0.378
500	55	2000	1.96	370	0.444
500	70	2500	2.45	440	0.528
1000	85	3500	3.43	525	0.63
1000	55	4500	4.41	580	0.696
1000	40	5500	5.39	620	0.744
1000	30	6500	6.37	650	0.480
1000	10	7500	7.35	660	0.792

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TABLE B-13:DISPLACEMENT TEST RESULTS(RUN 2)

	1	2		3	4	5
	<u></u> cc	cc	CC	PV		
	500	125	500	.48	125	0.15
	500	95	1000	.97	220	0.264
	500	75	1500	1.45	295	0.354
	500	75	2000	1.93	370	0.444
	500	70	2500	2.42	440	0.528
	1000	80	3500	3.38	520	0.624
	1000	55	4500	4.35	575	0.69
	1000	. 40	5500	5.32	615	0.738
	1.000	20	6500	6.29	635	0.762
	1000	5	7500	7.25	640	0.768

TABLE B-14:DISPLACEMENT TEST RESULTS

(RUN 3)

1	2	3		4	5
<u> </u>	cc	CC	PV	•	
720	168	720	.7	168	.201
500	78	1220	1.18	246	.295
500	74	1720	1.66	320	.384
500	60	2220	2.1.4	380	.455
500	57	2720	2.63	437	.525
980	83	3700	3.57	520	.625
1000	55	4700	4.54	575	.69
1000	20	5700	5.51	595	.714
1000	12	6700	6.47	607	.727
1000	Trace	7700	7.44	Trace	.73

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	1	2		3		5
	cc	cc	cc	PV		
	500	125	500	. 49	125	.153
	500	90	1000	.98	215	.263
	500	70	1500	1.47	285	.348
	500	65	2000	1.96	350	.428
	500	45	2500	2.45	395	.483
	L000	80	3500	3.43	475	.580
ן גי ר	1000	50	4500	4.42	525	.641
]	L000	25	5500	5.4	550	.672
]	L000	5	6500	6.38	555	.678
]	L000	~	7500	7.36	-	-

TABLE B-16:

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1	2		3		5
CC	CC	cc	PV		
500	115	500	0.49	115	.143
500	85	1000	0.97	200	.249
500	75	1500	1.46	275	.342
500	55	2000	1.95	330	.41
500	45	2500	2.43	375	.467
1000	80	3500	3.41	455	.567
1000	45	4500	4.38	500	.623
1000	20	5500	5.36	520	.647
1000	5	6500	6.33	525	.653
1000	-	7500	7.3	-	-

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140000 0-1/1	TABLE	B-17:
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DISPLACEMENT TEST RESULTS

(RUN 6)

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	1	2	3		4	5
	CC	cc	cc	PV		
	500	135	500	0.48	135	0.162
	500	80	1000	0.97	215	0.258
	500	70	1500	1.45	285	0.342
	500	50	2000	1.93	335	0.402
	500	50	2500	2.41	385	0.462
1	1000	75	3500	3.38	460	0.552
)	1000	40	4500	4.35	500	0.6
	1000	20	5500	5.31	520	0.624
	1000	5	6500	6.28	525	0.63
	1000	-	7500	7.25	-	-

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APPENDIX B2

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SUMMARY OF DISPLACEMENT TEST RESULTS

(PHASE 2)

TABLE B-21

PROPERTIES OF THE SANDPACK AND FLUIDS

Oil gravity		20°API
CO ₂ /steam ratio	=	.002 scF/cc steam inj.
Steam injection rate	=	30 cc/min. water eq.

Run no.	P.V.	N	S _{oi}	Τ _s
	cc	cc		°F
7	1031	833.36	.8083	300
8	1027	833.3	.8114	350
9	1032	818.4	.793	400
10	1023	818.4	.80	450
11	1019	787.9	.7732	500
12	1030	803.1	.7797	550

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1	2		3		5
cc	cc	cc	• PV		
500	145	500	0.48	145	0.174
500	140	1000	0.97	285	0.342
500	100	1500	1.45	385	0.462
500	75	2000	1.94	460	0.552
500	55	2500	2.42	515	0.618
1000	85	3500	3.39	600	0.72
1000	40	4500	4.36	640	0.768
1000	25	5500	5.33	665	0.798
1000	10	6500	6.3	675	0.81
1000	-	7500	7.27	-	

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TAB	L.E	B	23
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1	2		3		5
cc	CC	cc	PV		
500	140	500	. 0.49	140	0.168
500	130	1000	0.97	270	0.324
500	105	1500	1.46	375	0.45
500	65	2000	1.95	440	0.528
500	70	2500	2.43	510	0.612
1000	70	3500	3.41	580	0.696
1000	45	4500	4.38	625	0.75
1000	25	5500	5.36	650	0.78
1000	5	6500	6.33	655	0.786
1000	-	7500	7.3	-	-

1	2	-	3		5
cc	cc	cc	PV		
500	120	500	0.48	120	0.145
500	105	1000	0.97	225	0.275
480	110	1480	1.43	335	0.41
500	90	1980	1.92	425	0.52
1000	98	2980	2.89	523	0.64
1000	48	3980	3.86	571	0.7
990	33	4970	4.82	604	0.74
1000	10	5970	5.78	614	0.75
1000	-	6970	6.75	-	-
1000	-	7970	7.72	-	-

TABLE B-25:

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DISPLACEMENT TEST RESULTS

(RUN 10)

	1	2		3	4	5
CC	cc	CC	PV		<u></u>	
	500	127	500	0.49	127	0.155
	500	120	1000	0.98	247	0.3
	500	105	1500	1.47	352	0.43
	500	60	2000	1.96	412	0.5
	500	60	2500	2.44	472	0.577
1	1000	50	3500	3.42	522	0.638
•	1000	35	4500	4.4	557	0.68
	1000	15	5500	5.38	572	0.699
	1000	5	6500	6.35	577	0.705
	1000	-	7500	7.33	-	-

TABLE B-26:DISPLACEMENT TEST RESULTS(RUN 11)

	1	2	3		4	5
	<u> </u>	cc	cc	PV		
	500	125	500	0.49	125	0.159
	500	115	1000	0.98	240	0.305
	500	95	1500	1.47	335	0.425
	500	70	2000	1.96	405	0.514
	500	40	2500	2.45	445	0.565
	1000	40	3500	3.43	485	0.616
I	1000	35	4500	4.42	520	0.66
	1000	10	5500	5.4	530	0.673
	1000	5	6500	6.38	535	0.679
	1000	-	7500	7.36	-	-

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TABLE B-27:

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(RUN 12)

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1	2	3		4	5
<u> </u>	CC	cc	PV		
500	135	500	0.485	135	0.168
500	135	1000	0.97	270	0.336
500	80	1500	1.46	350	0.436
500	· 65	2000	1.94	415	0.498
500	45	2500	2.43	460	0.552
1000	70	3500	3.4	510	0.612
1000	30	4500	4.37	540	0.648
1000	15	5500	5.34	555	0.666
1000	5	6500	6.31	560	0.672
1000	-	7500	7.28	-	-

TABLE B-31

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PROPERTIES OF THE SANDPACK AND FLUIDS

0il Gravity = 20° API CO₂/steam ratio = .004 scF/cc. steam injected Steam injection rate = 30 cc/min. water eq.

Run no.	P.V.	N	S _{oi}	T _s
•	cc	cc	,,,,,	°F
13	1026	833.3	.8122	300
14	1023	848.5	.8294	350
15	1030	818.2	.7944	400
16	1021	787.9	.7717	450
17	1029	833,4	.81	500
18	1018	787.9	.774	550

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	1	2.	3		4	5
	cc	cc	CC	PV		
	500	210	500	0.49	210	.252
	500	180	1000	0.97	390 .	.468
	500	125	1500	1.46	515	.618
	500	60	2000	1.95	575	.69
	500	30	2500	2.44	605	.726
	1000	45	3500	3.41	650	.78
)	1000	25	4500	4.39	675	.81
	1000	5	5500	5.36	680	.816
	1000		6500	6.33	-	-
	1000	-	7500	7.31	-	-

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1	2	3		4	5
CC	cc	cc	PV		······
500	210	500	0.49	210	.247
500	195	1000	0.98	405	.477
500	120	1500	1.47	525	.619
500	60	2000	1.96	585	.689
500	25	2500	2.44	610	.719
1000	45	3500	3.42	655	.772
1000	20	4500	4.4	675	.795
1000	. 5	5500	5.38	680	.801
1000	-	6500	6.35	-	_

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1	2		3		5
<u> </u>	cc	cc	PV		
500	200	500	0.485	200	.245
500	175	1000	0.97	375	.458
500	105	1500	1.46	480	.587
500	50	2000	1.94	530	.648
500	35	2500	2.43	565	.69
1000	35	3500	3.4	600	.730
1000	20	4500	4.37	620	.758
1000	10	5500	5.34	630	.77
1000	-	6500	. 6.31	-	-

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(RUN 16)

1	2	·3		4	5
cc	cc	cc	PV	······································	
500	180	500	0.49	180	0.0.0
500	160	1000	0.98	340 .	. 228 . 431
500	120	1500	1.47	460	. 4 3 1
500	30	2000	1.96	490	.622
500	35	2500	2.45	525	.666
1000	30	3500	3.43	555	.704
1000	20	4500	4.41	575	.73
1000	5	5500	5.39	580	.736
1000	-	6500	6.37	-	

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(RUN 17)

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	1	· 2	3		4	5
	<u></u> cc	cc	CC	PV		
	500	185	500	0.49	185	. 222
	500	175	1000	0.97	360	.432
	500	105	1500	1.46	465	.558
	500	45	2000	1.94	510	.612
	500	35	2500	2.43	545	.654
152	1000	35	3500	3.4	580	.696
52	1000	10	4500	4.37	590	.71
	1000	5	5500	5.34	595	.714
	1000	-	6500	6.32	-	-

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	1	2	3		4	5
	СС	cc	cc	PV		
	500	230	500	0.49	230	.292
	500	160	1000	0.98	390	.495
	500	65	1500	1.47	455	.577
	500	25	2000	1.96	480	.609
	500	30	2500	2.46	510	.647
	1000	25	3500	3.44	535	.679
5	1000	10	4500	4.42	545	.692
	1000	. 5	5500	5.40	550	.698
	1000	-	6500	6.38	-	-

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TABLE B-41

PROPERTIES OF THE SANDPACK AND FLUIDS

Oil gravity	=	20°API
CO ₂ /steam ratio	=	.006 scF/cc. steam injected
Steam injection rate	=	30 cc./min. water eq.

Run no.	P.V.	N	S _{oi}	τ _s	
	cc	cc		°F	
19	1033	803	.7774	300	
20	1025	803	.7827	350	
21	1033	818.1	.792	400	
22	1029	803	.7804	450	
23	1031	833.3	.8083	500	
24	1022	803	.7858	550	

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1	2	3		4	5
CC	cc	cc	PV		
500	170	500	0.48	170	.212
500	150	1000	0.97	320	.398
500	100	1500	1.45	420	.523
500	85	2000	1.94	505	.629
500	55	2500	2.42	560	.697
1000	50	3500	3.39	610	.760
1000	30	4500	4.36	640	.797
1000	10	5500	5.32	650	.81
1000	-	6500 ·	6.29	-	_

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1	2	3		4	5
cc	cc	CC	PV		
500	160	500	0.49	160 ·	.2
500	155	1000	0.97	315	.392
500	120	1500	1.46	435	.542
500	70	2000	1.95	505	.629
500	45	2500	2.44	550	.685
1000	50	3500	3.41	600	.747
1000	25	4500	4.39	625	.778
1000	10	5500	5.36	635	. 79
1000	-	6500	6.33	-	-

(RUN 21)

1	2	3		. 4	5
cc	cc	cc	PV		
460	135	460	0.445	135	.165
500	150	960	0.93	285	.35
500	125	1460	1.41	410	.5
490	75	1950	1.89	485	.593
500	45	2450	2.37	530	.648
1000	50	3450	3.34	580	.71
1000	35	4450	4.31	615	.751
970	5	5420	5.25	620	.76
1000	-	6420	6.21	-	_

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(RUN 22)

1	2		3		5
CC	cc	cc	PV		
500	160	500	0.49	160	.199
500	130	1000	0.97	290	.361
500	115	1500	1.46	405	. 504
500	60	2000	1.94	465	.579
500	40	2500	2.43	505	.629
1000	50	3500	3.4	555	.691
1000	20	4500	4.37	575	.716
1000	5	5500	5.34	580	.722
1000	-	6500	6.32	-	

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(RUN 23)

1	2	3		4	5
<u> </u>	cc	CC	PV		
500	140	500	0.48	140	.168
500	145	1000	0.97	285	.342
500	125	1500	1.45	410	.492
500	80	2000	1.94	490	.588
500	35	2500	2.42	525	.63
1000	40	3500	3.39	565	.678
1000	15	4500	4.36	580	.696
1000	5	5500	5.33	585	.702
1000	-	6500	6.3	-	-

TABLE B-47: DISPLACEMENT TEST RESULTS

(RUN 24)

1	2 CC	3		4	5
<u>cc</u>		cc	PV		
500	200	500	0.49	200	.249
500	145	1000	0.98	345	.43
500	80	1500	1.47	425	.51
500	40	2000	1.96	465	.579
500	30	2500	2.45	495	.616
1000	35	3500	3.43	530	.66
1000	20	4500	4.40	550	.685
1000	5	5500	5.38	555	.69
1000	~	6500	6.36	-	-

APPENDIX C

SUMMARY OF DISPLACEMENT TEST RESULTS

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(PHASE 3)

PHASE 3

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TABLE C-11

PROPERTIES OF THE SANDPACK AND FLUIDS

Oil Gravity	= 20° API
CO ₂ /steam ratio	= .004 scF/cc steam injected
T _s	= 400°F

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Run no.	steam injection rate water eq. cc/min.	P.V.	N	S _{oi}
25	. 15	1027	803	.782
26	45	1023	818.4	.786
27	60	1026	803	.783

TABLE C-12DISPLACEMENT TEST RESULTS(RUN 25)

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1	2	:	3	4	5
CC	cc	CC	PV		
500	150	500	0.49	135	.168
500	130	1000	0.97	240	. 299
500	95	1500	1.46	335	.417
500	65	2000	1.95	410	.510
500	45	2500	2.43	460	.573
1000	55	3500	3.41	525	.654
1000	30	4500	4.38	560	.697
1000	10	5500	5.36	580	.722
1000	10	6500	6.33	585	.729
1000	-	7500	7.3	-	-

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TABLE C-13DISPLACEMENT TEST RESULTS(RUN 26)

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1	2		. 3		5
<u>cc</u>	cc	cc	PV		
500	160	500	0.49	160	.196
500	140	1000	0.97	300	.367
500	110	1500	1.46	410	.501
500	65	2000	1.95	475	. 58
500	50	2500	2.43	525	.641
1000	55	3500	3.41	580	.710
1000	25	4500	. 4.38	605	.739
1000	10	5500	5.36	615	.751
1000	-	6500	6.35	-	-

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(RUN 27)

1	2		3		, 5
cc	ec	cc	PV		
500	135	500	0.49	135	.168
500	130	1000	0.97	265	.330
500	105	1500	1.46	370	.461
500	75	2000	1,95	445	.554
500	45	2500	2.43	490	.610
1000	55	3500	3.41	545	.679
1000	20	4500	4.38	565	.704
1000	-	5500	5.36	-	

APPENSIX D

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SUMMARY OF DISPLACENT TEST RESULTS

(PHASE 4)

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TABLE D-11

PROPERTIES OF THE SANDPACK AND FLUIDS

	Steam injection	rate	=	30 cc/min.	(water eq.)		
	T _s		=	400°F			
Run	no. CO ₂ /steam ratio . scF/CO ₂ / steam cc.	P.V. cc		N CC	s _{oi}	Oil	gravity API
-28	0	1030		828.57	.8044	15	
29	.002	1025		828.57	.808	15	
30	.004	1027		828.57	.807	15	
31	.006	1033		828.57	.802	15	
32	0	1013		726.32	.717	26	
33	.002	1009		708.3	.702	26	
34	.004	1013		717.2	.708	26	
35	.006	1014		701.7	.692	26	

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TABLE D-12:

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1	2	3		4	5
CC	CC	CC	PV		
500	08	500	0.48	80	.0965
500	55	1000	0.97	135	.1630
500	50	1500	1.46	185	.2232
500	45	2000	1.94	230	.2776
500	35	2500	2.43	265	.3198
1000	65	3500	3.4	330	.3983
1.000	45	4500	4.37	375	.4526
1000	25	5500	5.34	400	.4827
1000	30	6500	6.31	430	.5189
1000	15	7500	7.28	445	.5371
1000	5	8500	8.25	450	.5431

1	2		3		5
cc	CC	cc	PV		
500	100	500	0.49	100	.1207
500	95	1000	0.98	195	.2353
500	75	1500	1.46	270	.3259
500	50	2000	1.95	320	.3862
500	30	2500	2.44	350	.4224
1000	50	3500	3.41	400	.4827
1000	35	4500	4.39	435	.5250
1000	20	5500	5.37	455	.5491
1000	15	6500	6.34	470	.5672
1000	5	7500	7.32	475	.5732
1000	-	8500	8.29	-	-

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	1	2	3		4	5
-	cc	CC	cc	PV		
	500	155	500	0.48	155	.1871
	500	125	1000	0.97	280	. 3379
	500	75	1500	1.45	355	.4284
	500	50	2000	1.94	405.	.4888
	500	25	2500	2.42	430	.519
L	500	25	3000	2.90	455	.5491
 ,	1000	35	4000	3.87	490	.591
	1000	15	5000	4.84	505	.609
	1000	10	6000	5.81	51.5	.621
	1000	-	7000	6.78	-	-

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1	2	:	3		5
CC	CC	cc	PV	<u></u>	
500	155	500	0.49	155	.1871
500	100	1000	0.97	255	. 3077
500	75	1500	1.46	330	. 3983
500	45	2000	1.95	375	. 4526
500	30	2500	2.43	405	.4888
1000	50	3500	. 3.41	455	.5491
1000	25	4500	4.38	480	. 5793
1000	15	5500	5.35	495	.5974
1000	5	6500	6.33	500	.6034
1000	-	7500	7.30	-	-

TABLE D-16:DISPLACEMENT TEST RESULTS(RUN 32)

1	2		3		5
<u> </u>	cc	cc	· PV		
500	130	500	. 495	130	.179
500	110	1000	.987	240	.330
500	95	1500	1.481	335	.461
500	85	2000	1.974	420	.578
500	50	2500	2.468	470	.647
1000	65	3500	3.455	535	.737
1000	35	4500	4.442	570	.785
1000	15	5500	5.429	585	.805
1000	5	6500	6.417	590	.812
1000	-	7500	7.404	-	-

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TABLE D-17:

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DISPLACEMENT TEST RESULTS

(RUN 33)

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2	3		4	5
cc	CC	PV		
140	500	0.495	140	.198
130	1000	0.991	270	.381
105	1500	1.487	375	. 529
80	2000	1.982	455	.642
40	2500	2.478	495	.7
50	3500	3.469	545	.769
25	4500	4.46	570	.805
1.0	5500	5.451	580	.82
	6500	6.442	-	-
	cc 140 130 105 80 40 50 25 10	cc cc 140 500 130 1000 105 1500 80 2000 40 2500 50 3500 25 4500 10 5500	cccc PV 140500 0.495 1301000 0.991 1051500 1.487 802000 1.982 402500 2.478 503500 3.469 254500 4.46 105500 5.451	$\begin{array}{c c c c c c c c c c c c c c c c c c c $

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TABLE D-18:

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DISPLACEMENT TEST RESULTS

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(RUN 34)

	1	2		3		5
	cc	cc	cc	PV		
	500	160	500	0.495	160	. 223
	500	150	1000	0.987	310	.432
	500	115	1500	1.481	425	. 592
	500	65	2000	1.974	490	.683
	500	40	2500	2.468	530	.739
L L	1000	35	3500	3.455	565	. 788
•	1000	15	4500	4.442	580	.81
	1000	-	5500	5.429	-	-

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TABLE D-19:DISPLACEMENT TEST RESULTS(RUN 35)

1	2		3		5
cc	CC	cc	PV		
500	155	500	0.495	155	.221
500	130	1000	0.987	285	.406
500	110	1500	1.481	395	.563
500	70	2000	1.974	465	.663
500	40	2500	2.468	505	.720
1000	40	3500	3.455	545	.777
1000	15	4500	4.442	560	.798
1000	5	5500	5.429	565	.805
1000	-	6500	6.41	-	-

APPENDIX E

SUMMARY OF DISPLACEMENT TEST RESULTS

(PHASE 5)

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TABLE E-11

PROPERTIES OF THE SANDPACK AND FLUIDS

Oil Gravity	=	20° API
Steam injection rate	=	30 cc/min. (water eq.)
Steam temperature	=	400°F
рH	=	12

Run no.	CO ₂ /steam ratio	P.V.	N	S _{oi}
	scF CO ₂ /cc steam	cc	cc	
•	•			
36	0	1023.5	825	.806
37	.002	1026.5	826	.8046
38	.004	1025.5	826	.8052
39	.006	1021.6	814	.797

TABLE E-12:DISPLACEMENT TEST RESULTS(RUN 36)

1	2		3		5
<u>cc</u>	cc	CC	PV .		
500	125	500	0.49	125	.151
500	95	1000	0.98	220	.267
500	80	1500	1.47	300	.364
500	70	2000	1.95	370	.448
500	60	2500	2.44	430	.521
1000	90	3500	3.42	520	.630
1000	60	4500	4.4	580	.700
1000	20	5500	5.37	600	.730
1000	10	6500	6.35	610	.740
1000	-	7500	7.31	-	-

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TABLE E-13: DISPLACEMENT TEST RESULTS

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(RUN 37)

1	2		3		5
cc	cc	CC	PV		<u></u>
500	125	500	0.49	125	.151
500	115	1000	0.98	240	.290
500	90	1500	1.47	330	.400
500	95	2000	1.95	425	.515
500	60	2500	2.44	485	. 587
1000	75	3500	3.42	560 [.]	.678
1000	45	4500	4.40	605	.732
1000	. 15	5500	5.37	620	.751
1000	-	6500	6.33	-	-

.

TABLE E-14: DISPLACEMENT TEST RESULTS

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(RUN 38)

1	2		3		5
CC	сс	cc	PV		
500	215	500	0.49	215	.260
500	200	1000	0.98	415	.502
500	80	1500	1.47	495	.599
500	45	2000	1.95	540	.654
500	30	2500	2.44	570	.670
1000	45	3500	3.42	615	.744
1000	15	4500	4.4	630	.763
1000	5	5500	5.37	635	. 769
1000	-	6500	6.34	-	_

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TABLE E-15: DISPLACEMENT TEST RESULTS

(RUN 39)

1	2	3		4	5
CC	cc	cc	PV		
500	205	500	0.49	205	.200
500	165	1000	0.98	370	.362
500	150	1500	1.47	520	.509
500	95	2000	1.95	615	.602
500	65	2500	2.44	680	.666
1000	55	3500	3.42	735	.719
1000	30	4500	4.4	765	.749
1000	10	5500	5.38	775	.759
1000	-	6500	6.36	-	-

APPENDIX F SUMMARY OF TEMPERATURE DISTRIBUTION RESUTLS WITH DISTANCE

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 RUN	TIME HOURS	0	DISTANCE FROM	1 INLET, 7	INCHES	19	
	1	300	300	300	245	115	
1	2	300	300	300	300	210	
	4	300	300	300	300	260	
	1	350	350	340	270	130	
2	2	350	350	350	340	230	
	4	350	350	350	350	280	
	1	400	400	365	290	140	
3	2 .	400	400	400	360	255	
	4	400	400	400	400	325	

TABLE F-1:	TEMPERATURE	DISTRIBUTION	WITH	DISTANCE.

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RUN	TIME HOURS	0	DISTANCE FROM	INLET, 7	INCHES	19
	1	300	300	300	245	115
1	2	300	300	300	300	210
	4	300	300	300	300	260
	1	350	350	340	270	130
2	2	350	350	350	340	230
	4	350	350	350	350	280
	1	400	400	365	290	140
3	2	400	400	400	360	255
	4	400	400	400	400	325

TABLE F-2: TEMPERATURE DISTRIBUTION WITH DISTANCE.

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1

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3
     This programme calaculates the temperature distribution in the
C
     porous medium
C
     ts=Steam injection temperature
Ξ
     rc1=Volumetric heat capacity oof porous medium
C
     rc2=volumetric heat capacity of suroundings
C
     rcf=volumetric heat of fluids
C
     rcr=volumetric heat capacity of surroundings
C
    k1=Thermal conductivity of porous medium
C
    k2=Thermal conductivity of surroundings
C
    h=Peservoir thickness
3
    t0=Initial reservoir temperature
2
    z=vertical listance from center of the reservoir
        read*,rcf,rc1,rcr,k1,k2,t0,ts,rc2,h,v,z,1
        10 50 ts=300,550,50
        print*, '-----
     1-----+
        print*,*
                          Injection Temperature= ',ts,' ieg.f'
        print*, '-----
     1 ---
        . . . . . . . . . . . . . . . . . 1
        do 10 t=.25,4,.25
        10 20 x=.25,2,.25
        theta=rc1/rcr
        eta=4*k2*x/((h**2)*rcf*v)
        tau=4*k2*t/((h**2)*ro1)
        if ([tau-eta].le.0.0) goto 10
        any=2*z/h
        a=(eta+abs(any)-1)/(2*sgrt(theta*(tau-eta)))
        x = x/1
        temp=t0+erfc(a) * (ts-t0)
        write(6,100) t,x,x1,erfc(a),temp
100
        format(3x, f5.2, 3x, f10.5, 3x, f10.5, 3x, f10.5, 3x, f10.5)
20
        continue
10
        continue
50
        continue
        stop
        end
        function erfc(x)
        implicit real (a-h, o-z)
        a1=.254825952
        a2=-.284496736
        a3=1,421413741
        a4=-1.453152027
        a5=1.061405429
```

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```
s=abs(x)
      t=1./(1.+.3275911*s)
      if (s-1e-30) 1,2,?
1
      sa=1.
      goto 3
2
      sa=s/x
3
      erf=t*(a1+t*(a2+t*(a3+t*(a4+a5*t)))) *exp(-x*x)
      erf=sa*(1.-erf)
      erfc=1-erf
      return
      end
62.4,42.45,36.3,1.4,1.4,105,400,36.3,2215567,1.2949,11078,2
```

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t hours	x ft.	x/l	Τ _D	T°F	
* - •	Injection Temp	erature=	.300000000e+03	lej.f	-
. 25	.25000	.12500	.72976	247.30292	-
.50	.25000	.12500	.84462	269.70114	
.50	.50000	.25000	.62518	226.91049	
.50	.75000	.37500	.27285	158_20490	
.75	.25000	.12500	.87975	276.55038	
.75	.50000	.25000	.73317	247.96797	
.75	.75000	.37500	.54964	212.17911	
.75	1.00000	.50000	.31527	166.47853	
.75	1.25000	.62503	.05144	115.03131	
1.00	.25000	.12500	-89840	280.18863	
1.00	.50000	.25000	.78166	257.42330	
1.00	.75000	.3750)	.64654	231.07565	
1.00	1.00000	.50000	.48965	200.48256	
1.00	1.25000	.62500	.30991	165.43236	
1.00	1.50000	.75000	.12097	128.53916	
1.00	1.75000	. 87500	.00335	105-65365	
1.25	.25000	.12500	.91042	232.53149	
1.25	.50000	.25000	.81071	263.08844	
1.25	.75000	.37500	.69952	241.42522	
1.25	1.00000	.50000	.57607	217.33453	
1.25	1.25000	.62500	.43983	190.77602	
1.25	1.50000	.7500 0	. 29353	162.23859	
1.25	1.75000	.87500	.14746	133.75406	
1.25	2.00000	1.00000	.03298	111.43024	
1.50	-25000	.12500	.91893	234.20114	
1.50	.50000	.25000	.83059	266.96420	
1.50	.75000	.37500	.73428	248.18414	
1.50	1.00000	.50000	-62972	227.79623	
1.50	1.25000	.62500	.51707	205-82816	
1.50	1.50000	.75000	.39749	182-50964	
1.50	1.75000	.87500	.27435	158.49828	
1.50	2.00000	1.00000	.15556	135_33427	
1.75	.25000	.12500	.92548	285.46838	
1.75	.50000	.25000	-84528	269.82980	
1.75	.75000	.37500	.75918	253.04019	
1.75	1.00000	.50000	.66712	235.08905	
1.75	1.25000	.62500	.56936	216-32464	
1.75	1.50000	.75000	.46666	195-99951	
1.75	1.75000	.37500	.36076	175.34869	
1.75	2.00000	1.00000	.25498	154.72093	
2.00	.25000	.12500	.93063	286.47266	
2.00	- 50000	.25000	.85671	272. 35936	
2.00	.75000	.37500	.77818	256.74561	
2.00	1.00000	.50000	.69510	240-54358	
2.00	1.25000	.62500	.60772	223.50583	

200 200 200 200 200 200 200 200	90.2380300 90.23805 90.2385 90.2585 90.2585 90.2585 90.2585
↓ 0 t 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
	25000 20000 20000 20000 20000 20000 20000 20000 20000 20000 20000 20000 20000 20000 20000 2000000
1 50000 2 50000 2 50000 2 50000 2 50000 2 50000 3 50000 3 50000 4 50000 4 50000 4 50000 4 50000 4 55000 4 55000 5 55000 4 55000 4 550000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5 55000 5	20000000000000000000000000000000000000
20000000000000000000000000000000000000	

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3.75	.50000	- 25000	-89859	230.22504	
3.75					
	.75000	- 37500	-84550	269.87231	
3.75	1.00000	.50000	.79097	259.23863	
3.75	1.25000	.62500	.73515	248.35518	
3.75					
	1.50000	.75000	.67826	237.26758	
3.75	1.75000	.87 500	.62055	226.00731	
3.75	2.00000	1.00000	.56232	214.55289	
4.00	. 25000				
		- 12500	-95175	290.59213	
4 - 00	.50000	. 25000	-90202	280.39444	
4_00	.75000	.37500	.85089	270.32273	
4.00	1.00300	.50000			
			-79846	260.69958	
4 - 60	1.25000	. 62500	.74489	250.25372	
4.00	1.50000	.75000	. 69036	239-52090	
4.00	1.75000	.87500	.63510	228.34494	
4.00	2.00000				
4 • 00	2.00000	1.00000	. 57938	217.97852	
	Injection Ten	perature=	.35000000e+03	B deg.f	
• 25	.25000	12500	7007(
		.12500	-72976	283.79083	
- 50	.25000	.12500	-94462	311.93219	
. 50	.50000	.25000	.62518	258.16959	
• 50	.75000	.37500	.27235	171.84718	
. 75					
	-25000	.12500	.87975	320.53765	
. 75	.50000	.25000	.73317	284.52643	
• 75	.75000	.37500	. 54964	239.66092	
. 75	1.00000	-50000	.31527		
. 75				182.24231	
	1.25000	.62590	-05144	117.60344	
1.00	.25000	.12500	_ 39840	325.10880	
1_00	.50000	-25000	.78166	296.50684	
1.00	.75000	.37500	.64654		
				263.40274	
1_00	1.00000	.50000	-48965	224.96527	
1.00	1.25000	.62500	.30991	180.92783	
1.00	1.50000	.75000	.12097	134.63766	
1.00	1.75000	. 37500			
			.00335	105.32125	
1.25	-25000	.12 500	- 91042	328.05237	
1.25	- 50000	-25000	.81071	303.62393	
1.25	.75000	.37500	.69962	276.40604	
1.25	1.00000	.50000	.57607		
				246.13826	
1.25	1.25000	-62500	- 43988	212.76985	
1.25	1.50000	-75000	.29353	176.31516	
1.25	1.75000	.87500	. 14746	141.12689	
1.25	2.00000	1.00000			
			-03298	113.07903	
1.50	.25000	.12500	.91698	330.15015	
1.50	.50000	.25000	.33059	308.49350	
1.50	.75000	.37500	.73428	294.89804	
1.50	1.00000	.50000			
			.52972	259.28244	
1.50	1.25000	.62500	.51707	231.68153	
1.50	1.50000	.75000	.39749	202.33.391	
1.50	1.75000	.87500	.27435	172.21579	
1.50	2.00000	1.00000	.15556		
			•+2220	143.11229	

1.75	.25000	.12500	.92548	331.74231
1.75	.50000	.25000	.84528	
				312.09387
1.75	.75000	.37500	.75918	29 0. 9992 1
1.75	1.00000	.50000	.65712	268.44522
1.75	1.25000			
		.62500	. 56936	244.49251
1.75	1,50000	.75000	.46656	219.33272
1.75	1.75000	.87500	.36076	193.3 8633
1.75	2.00000	1.00000	.25493	167.46989
2.00	.25000	.12500	.93063	333.00409
2.00	.50000	.25000	. 35671	314.89508
2.00	.75000	.37500	.77818	295.65475
2.00	1.00000	.50000		
			.69510	275.29834
2.00	1.25000	.62500	<u>.</u> 60772	253.89194
2.00	1.50000	.75000	.51666	231.58049
2.00	1.75000	.87500	.42298	208.63129
2.00	2.00000	1.00000	. 32856	185.49789
2.25	. 25000	- 12500	.93484	334.03589
2.25	.50000	.25000	.86594	317.15494
2.25	.75000	.37500	.79330	299.35791
2.25	1.00000	- 50000	.71703	230.57245
2.25	1.25000	.62500	-63740	261.15281
2.25	1.50000	.75000	- 55488	240.94550
2.25	1.75000	.87500	-47025	220.21147
2.25	2.00000	1.00000	.38472	199.25601
2.50	.25000	.12500	.93837	
				334.89999
2.50	.50000	.25000	.87358	319.02792
2.50	.75000	.37500	-80569	302.39456
2.50	1.00000	.50000	.73483	285.03259
2.50	1.25000	-62500		
			.66123	267.00208
2.50	1.50000	.75000	. 58531	248.39999
2.50	1.75000	.97500	.50764	229.37269
2.50	2.00000	1.00000	.42911	210.13153
2.75	.25000	. 12500	-94138	335.63745
2.75	. 50000	- 25000	-88005	320.61316
2.75	.75000	.37500	-81609	304.94327
2.75	1.00000	.50000	.74564	238.56199
2.75	1.25000	.62500	-68032	271.32462
2.75	1.50000	.75000	.61026	254.51372
2.75	1.75000	.87500	.53815	236.34666
2.75	2.00000	1.00000	- 46524	
				218.98462
3.00	.25000	.12500	•94399	336.27646
3.00	. 50000	- 25000	.88562	321.97751
3.00	.75000	.37500	.82499	· 307.12213
3.00	1.00000	- 50000	-76222	291.74423
3.00	1.25000	.62500	-69753	275.39453
3.00	1.50000	.75000	-63120	259.54456
3.00	1.75000	.87500	.56364	243.09143
3.00	2.00000	1.00000	.49536	226.35348
3.25	.25000	.12500	.94€27	336.83710

3.25	.50000	.25000	.89048	323.16794	
3.25	.75000	.37500	.93270	339.31266	
			.77308	294.40424	
3.25	1-00000	- 50000			
3.25	1.25000	.62500	.71179	279.38904	
3.25	1.50000	.75000	.64910	264.02988	
3.25	1.75000	.87500	.58534	248.40906	
3.25	2.00000	1.00000	.52095	232.63197	
3.50	.25000	.12500	• 94 R 30	337.33420	
3.50	.50000	.25000	.89477	324.21848	
3.50	.75000	.37500	.83948	310.67334	
3.50	1.00000	.50000	.76257	296.73038	
			.72421	292.43204	
3.50	1.25000	.62500			
3.50	1.50000	.75000	.66463	267.83401	
3.50	1.75000	.87500	-63411	253.00743	
3.50	2.00000	1.00000	.54302	238.04089	
3.75	.25000	.12500	.95012	337.77893	
3.75	.50000	.25000	-89859	325.15454	
3.75	.75000	.37500	.34550	312.14728	
3.75	1.00000	.50000	.79097	298.78705	
3.75	1.25000	.62500	.73515	285.11292	
				271.17480	
3.75	1.50000	.75000	.67826		
3.75	1.75000	.87500	.62055	257.03482	
3.75	2.00000	1.00000	.56232	242.76903	
4.00	-25000	.12500	.95175	338.17984	
4.00	- 50000	.25000	.90202	325.99557	
4.00	.75000	.37500	-85089	313.46704	
4.00	1.00000	.50000	.79846	300.62256	
4.00	1.25000	.62500	.74439	287.49826	
4.00	1.50000	.75000	.69036	274.13907	
4_00	1.75000			260.50007	
		.87500	.63510		
4_00	2.00000	1.00000	.57938	246.94736	

	Injection Temp	erature=	.4000000000e+03	deg.f	
.25	- 25000	.12500	.72576	320.27878	
. 50	.25000	.12500	-84462	354.15327	
.50	.50000	.25000	.62518	239.42368	
- 50	.75000	.37500	.27285	185.48946	
	.25000	.12500	.87975	364.52496	
• 75				321.28488	
. 75	- 50000	.25000	.73317		
.75	.75000	.37500	-54964	267.14273	
.75	1_00000	.50000	.31527	198.00604	
. 75	1.25000	.62500	-05144	120.17558	
1.00	.25000	.12500	-89840	370_02896	
1.00	- 50000	.25000	.78166	335.58987	
1.00	.75000	. 37500	.64654	295.72983	
1.00	1.00000	.50000	.48965	249.44798	
1_00	1.25000	.62500	.30991	196.42331	
1_00	1.50000	.75000	.12097	140.58617	
			.00335	105.99885	
1.00	1.75000	.87500	.00333	-030000	

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		•		
1.25	.25000	-12500	.91042	373.57327
1.25	. 50000	.25000	.81071	344.15942
			.69962	311.33637
1.25	.75000	- 37500		
1.25	1.00000	- 50000	. 57607	274.94195
1.25	1.25000	.62500	.43988	234.76370
1.25	1.50000	.75000	.29353	191.59172
1.25	1.75000	.87500	.14746	148.49973
1.25	2.00000	1.00000	.03298	114.72781
1.50	.25000	12500	. 91898	376.09915
1.50	.50000	<u>- 25000</u>	.83059	350.02277
1.50	.75000	.37500	.73428	321.61191
1.50	1.00000	. 50000	.62972	290.76865
		.62500	.51707	257.53491
1.50	1.25000			
1.50	1.50000	-75000	.39749	222.25318
1.50	1.75000	.87 500	.27435	185.93330
1.50	2.00000	1_00000	. 15556	150.89030
1.75	.25000	.12500	- 92548	378.31627
1.75	.50000	.25000	-84528	354.35791
1.75	.75000	- 3750)	.75913	328.95925
1.75	1.00000	.50000	.65712	301.30139
1 .7 5	1.25000	.62500	.56936	272.96036
1.75	1.50000	- 75000	.46666	242.65592
1.75	1.75000	-87500	.35076	211.42494
1.75	2.00000	1.00000	-25493	180.21884
2.00	.25000	-12500	.93063	379.53555
2.00	- 50000	. 25000	. 85671	357.73093
2.00	.75000	.37500	.77813	334.56387
2.00	1.00000	- 50000	.69510	310.05310
2.00	1.25000	.62500	.60772	284.27805
2.00	1.50000	.75000	.51666	257.41324
2.00	1.75000	.87500	-42298	229.78053
2.00	2.00000	1_00000	-32856	201.92604
2.25	.25000	.12500	.93434	380.77789
2.25	.50000	- 25000	.86594	360.45184
2.25	.75000	.37500	.79330	339.02280
2.25	1.00000	. 50000	.71703	316.52399
2.25	1.25000	.62500	.63740	293.03278
2.25	1.50000	- 75000	-55438	268.68945
2.25	1.75000	.87500	.47025	243.72401
2.25	2.00000	1_00000	.38472	215.49193
2.50	.25000	.12500	.93837	381.81936
2.50	.50000	. 25000	.87358	362.70709
		.37500		342.57917
2.50	.75000		-80569	342.37317
		C D C A C A		
2.50	1.00000	.50000	.73433	321.77396
2.50	1.25000	<u>.</u> 62500	.66123	300.06372
2.50	1.50000	.75000	.53531	277.50531
2.50	1.75000	.87500	.50764	254.75495
2.50	2.00000	1.00000	.42911	231.58694
			.94138	382.70633
2.75	.25000	- 12500		
2.75	.50000	- 25000	.88005	364.61584

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2.75	.75000	. 37500	.81609	345.74802	
2.75	1.00000	-50000	.74964	326.14401	
2.75	1.25000	- 62500 - 62500	.63092	305.97045	
		.75000	.61026		
2.75	1.50000			285.02673	
2.75	1.75000	. 87500	-53815	263.75415	
2.75	2.00000	1_00000	.46524	242.24678	
3.00	.25000	. 12500	.94399	383.47574	
3.00	.50000	. 25000	.88562	366.25861	
3.00	.75000	. 37500	.82499	348.37155	
3.00	1.00000	.50000	. 7£222	329.35529	
3.00	1.25000	.62500	.69753	310.77097	
3.00	1.50000	.75000	.63120	291.20468	
3.00	1.75000	.87500	- 56364	271.27335	
3.00	2.00000	1.00000	.49536	251.13155	
3.25	.25000	. 12500	.94627	384.15079	
3.25	.50000	.25000	. 99048	367.59202	
3.25	.75000	. 37500	.83270	350.64792	
3.25	1.00000	-50000	.77308	333.05817	
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3.25	1.75000	.87500	.58534	277.67621	
3.25	2.00000	1_00000	.52095	258.57932	
3.50	.25000	- 12500	-94830	384.74933	
3.50	-50000	-25000	.89477	368.95694	
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3.50	1.00000	.50000	.79257	335.85901	
3.50	1.25060	<u>-</u> 62500	. 72421	318.64264	
3.50	1.50000	.75000	.66463	301.06546	
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3.50	2.00000	1.00001	.54302	265.19211	
3.75	.25000	. 12500	.95012	385.28482	
3.75	.50000	.25000	.39859	370.08405	
3.75	.75000	.37500	.84550	354.42224	
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3.10	1.75000	.87500	.62055	288.06232	
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4.00	.25000	.12500	.95175	385.70758	
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25       25)00       12500       9048         25       50000       25000       37500       33270         25       1.00000       50000       37500       33270         25       1.25000       50000       9048         25       1.25000       50000       91020         25       1.25000       50000       91179         25       1.50000       62500       91179         25       1.75000       95000       94831         25       2.0000       1.00000       52095         250       2.0000       1.2500       94830	տ	8947	2500	5000	• ហ
25       25)30       12500       94627         25       50000       25000       9048         25       75000       37500       33270         25       1.00000       50000       77308         25       1.25000       62500       77308         25       1.25000       62500       71179         25       1.25000       75000       54910         25       1.75000       87500       54910         25       2.0000       1.00000       52095	ر.	53+6	1250	2500	• ហ
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.25       .25))0       .12500       .94627         .25       .50000       .25000       .9048         .25       .75000       .37500       .33270         .25       1.00000       .50000       .77308         .25       1.25000       .62500       .71179         .25       1.50000       .75000       .64910	ω	5853	750	7500	• 2
.25       .25))0       .12500       .94627         .25       .50000       .25000       .9048         .25       .75000       .37500       .33270         .25       1.00000       .50000       .77308         .25       1.25000       .62500       .71179	ω	1619	500	50000	• •
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	U:	735	250	25))	• N

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APPENDIX G

## PLOTS OF LABORATORY RESULTS

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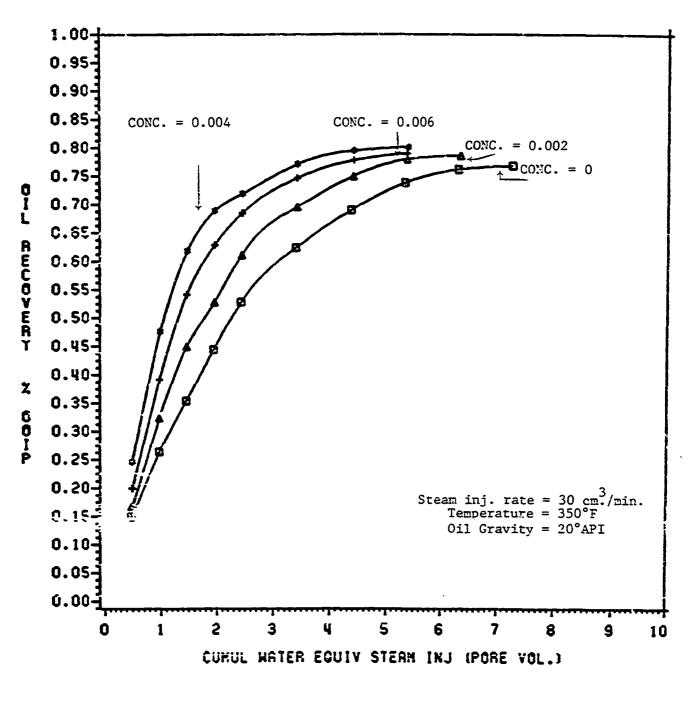


FIGURE G1: EFFECT OF  $CO_2$  ON STEAM DRIVE RECOVERY CONC. =  $CO_2$ /STEAM RATIO (SCF  $CO_2$ /CM³ STEAM INJ.)

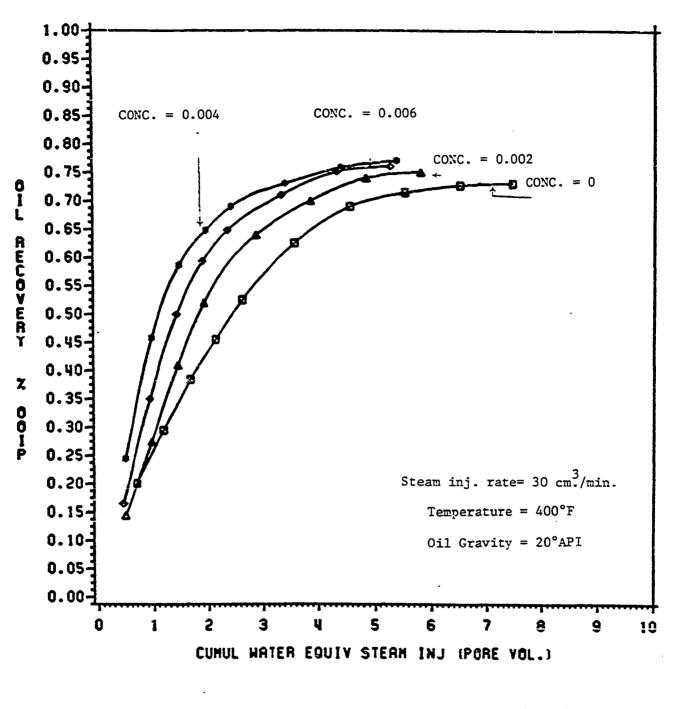
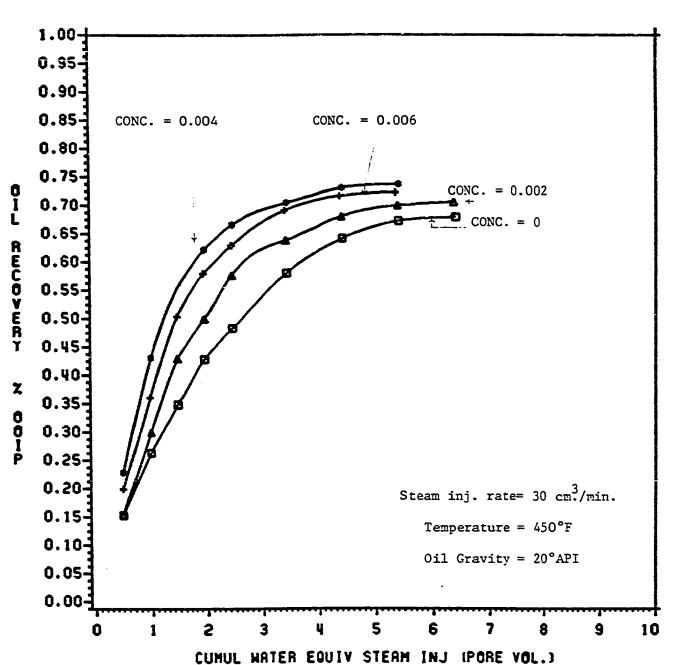


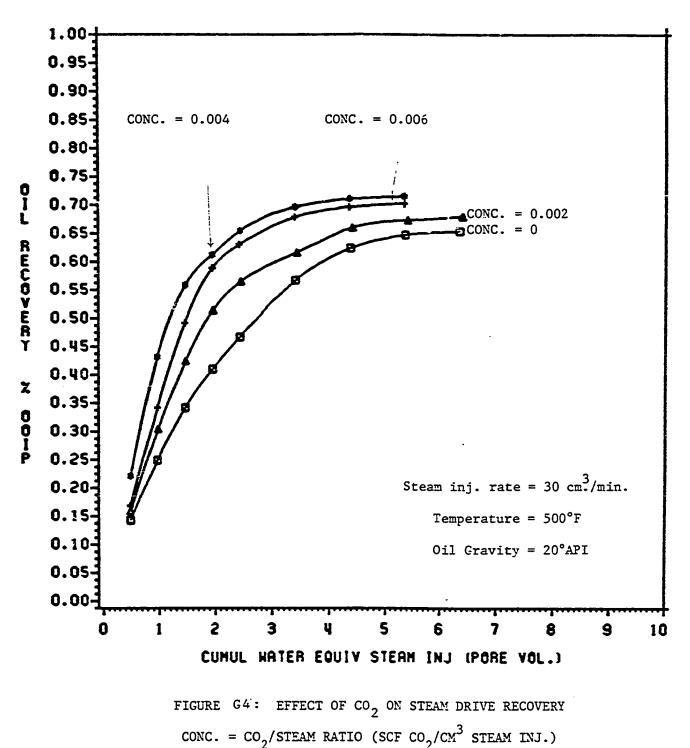
FIGURE G2: EFFECT OF  $CO_2$  ON STEAM DRIVE RECOVERY COMC. =  $CO_2$ /STEAM RATIO (SCF  $CO_2$ /CM³ STEAM INJ.)

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EFFECT OF CO2 ON STEAM DRIVE RECOVERY

FIGURE G3: EFFECT OF  $CO_2$  ON STEAM DRIVE RECOVERY CONC. =  $CO_2$ /STEAM RATIO (SCF  $CO_2$ /CM³ STEAM INJ.)



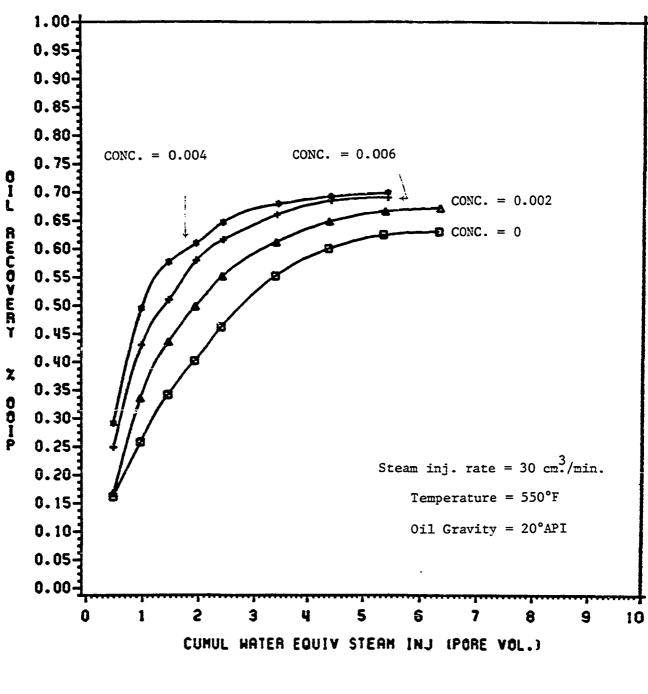
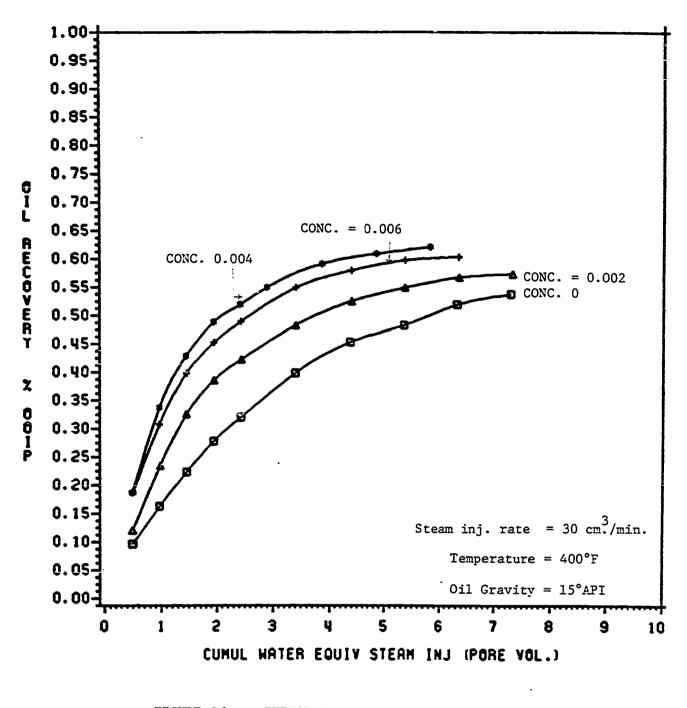
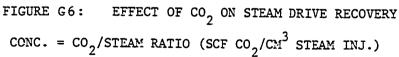
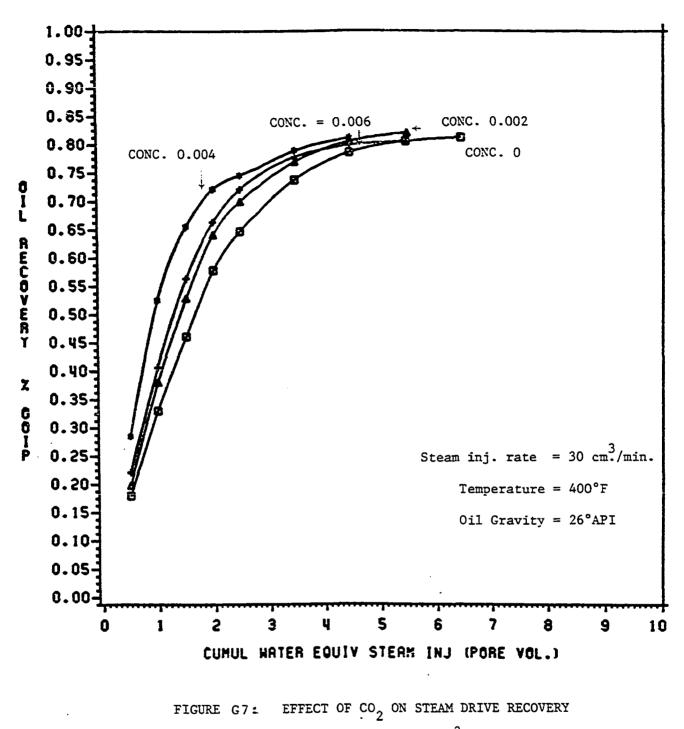


FIGURE G5: EFFECT OF  $CO_2$  ON STEAM DRIVE RECOVERY CONC. =  $CO_2$ /STEAM RATIO (SCF  $CO_2$ /CM³ STEAM INJ.)







CONC. =  $CO_2$ /STEAM RATIO (SCF  $CO_2$ /CM³ STEAM INJ.)

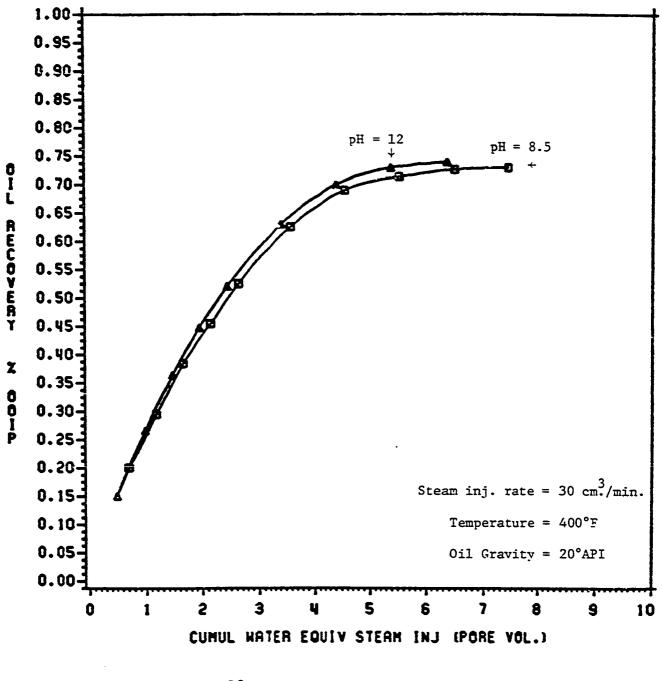
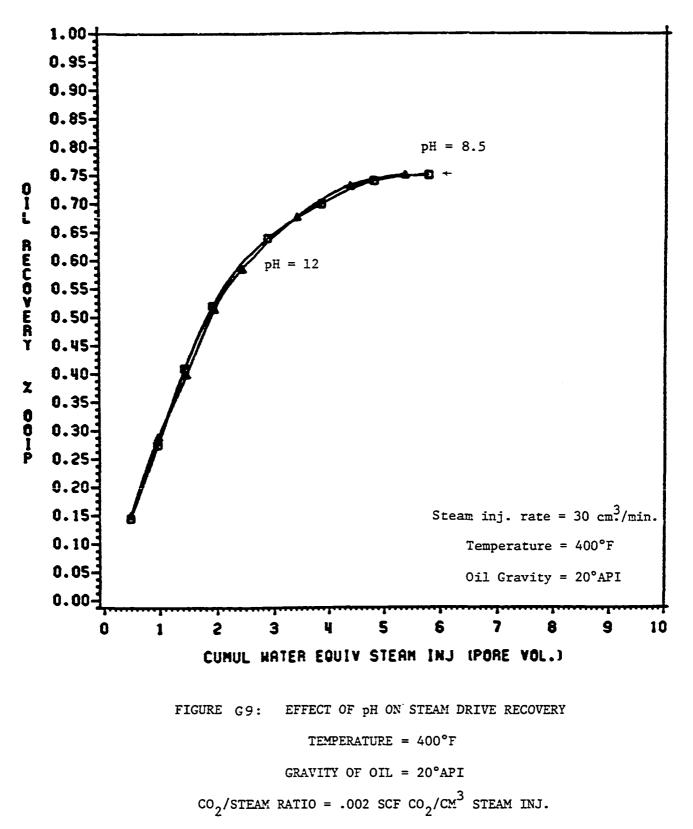
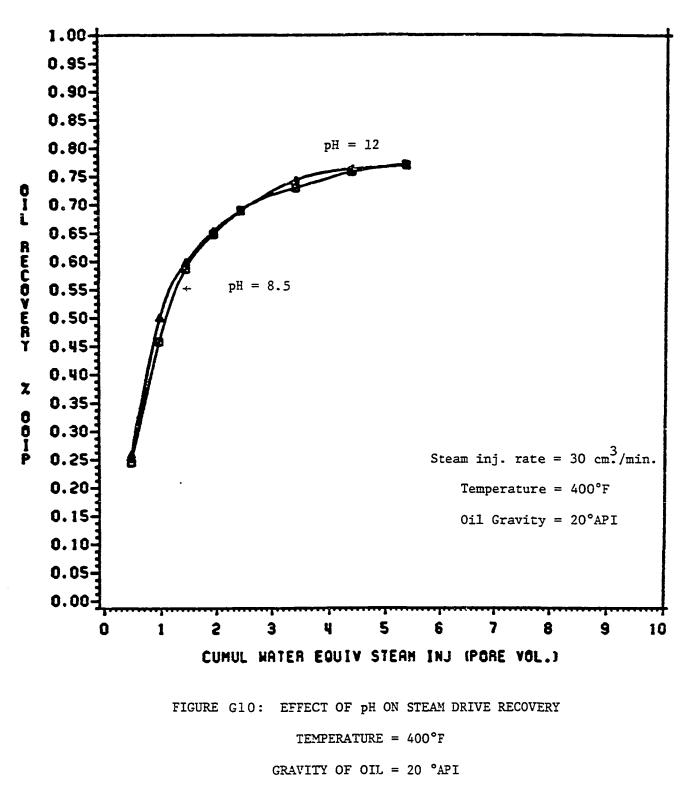


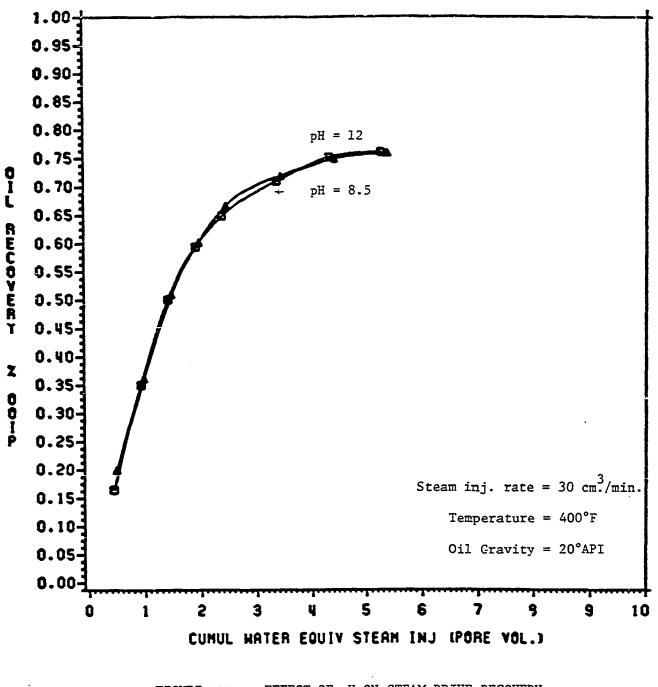
FIGURE G8: EFFECT OF pH ON STEAM DRIVE RECOVERY  $CO_2/STEAM$  RATIO = 0.00 SCF  $CO_2/CM^3$  STEAM INJ.

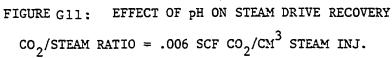




 $CO_2$ /STEAM RATIO = .004 SCF  $CO_2$ /CM³ STEAM INJ.

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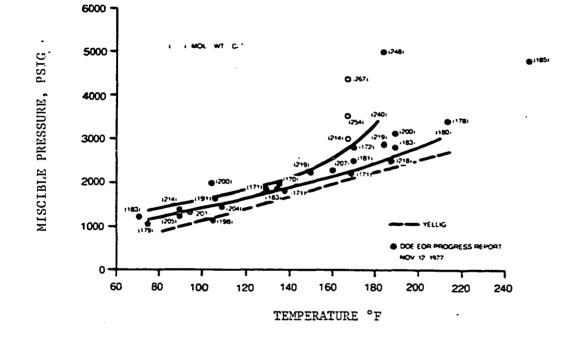


FIGURE G12: HOLM-JOSENDAL DYNAMIC MISCIBILITY DISPLACEMENT CORRELATION FOR CO₂ (AFTER REF. 67)

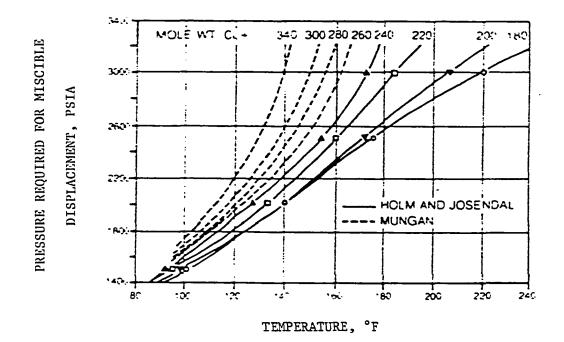
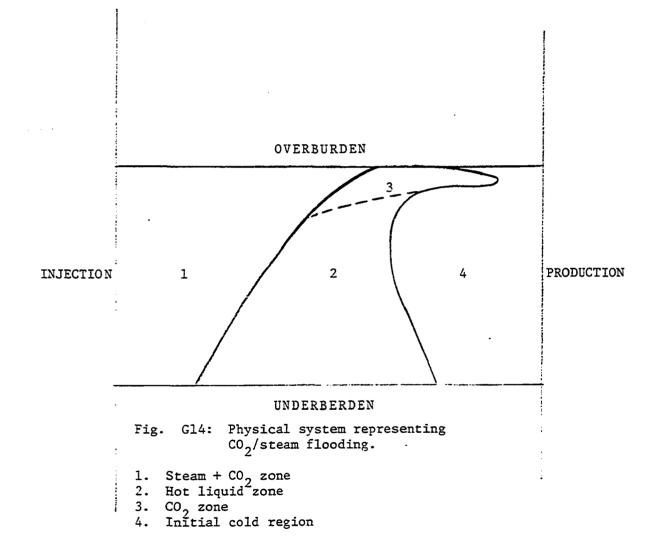


FIGURE G13: PRESSURE REQUIRED FOR MISCIBLE DISPLACEMENT IN CO₂ FLOODING (AFTER REF.68)



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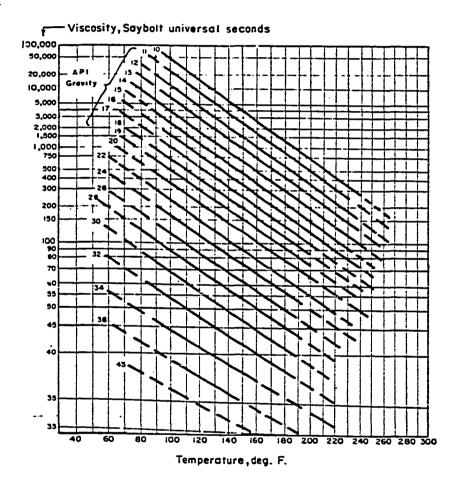
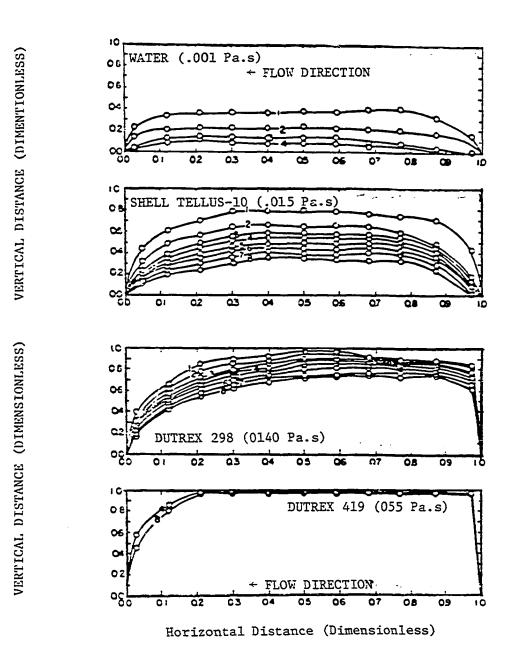
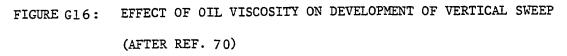


FIGURE G15: CHART FOR TEMPERATURE VARIATION OF CRUDE OIL VISCOSITY

(AFTER REF.69)

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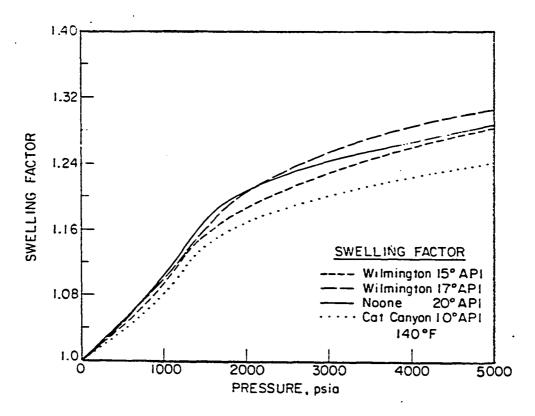


FIGURE G17: Swelling factor comparison of heavy oils at 140°F (after Ref. 71)

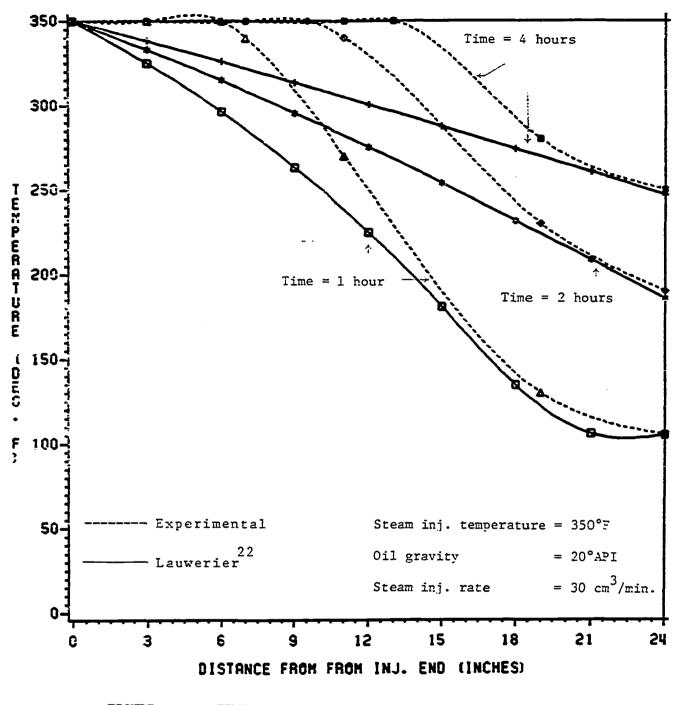


FIGURE G18: TEMPERATURE DISTRIBUTION WITH DISTANCE FOR RUN NO. 2

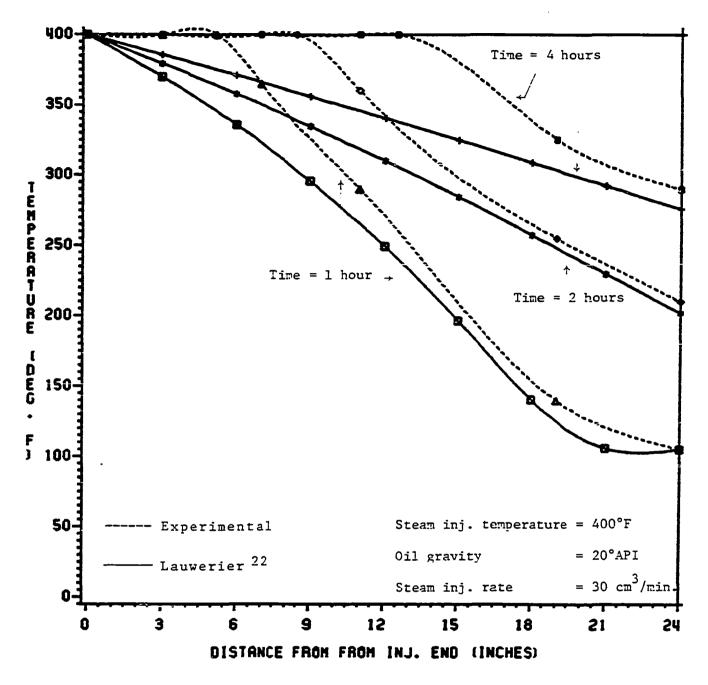


FIGURE G19: TEMPERATURE DISTRIBUTION WITH DISTANCE FOR RUN NO. 3

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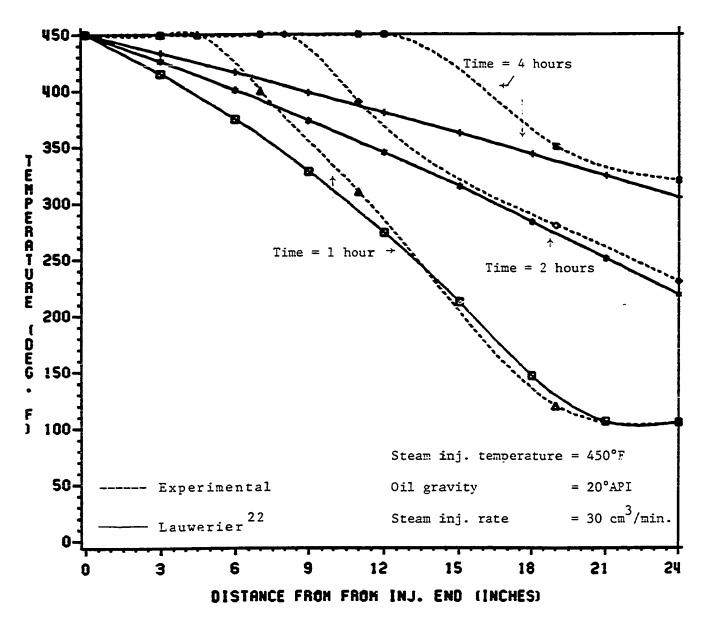


FIGURE G20: TEMPERATURE DISTRIBUTION WITH DISTANCE FOR RUN NO. 4

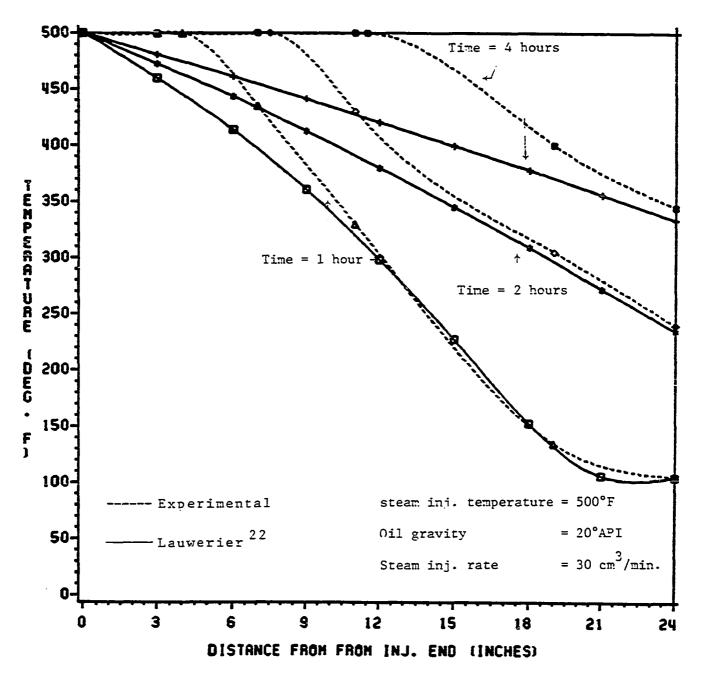


FIGURE G21: TEMPERATURE DISTRIBUTION WITH DISTANCE FOR RUN NO. 5

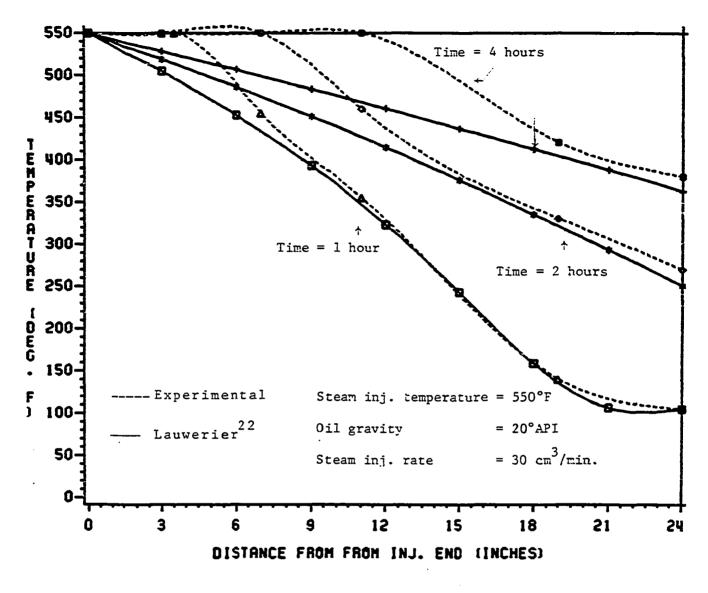
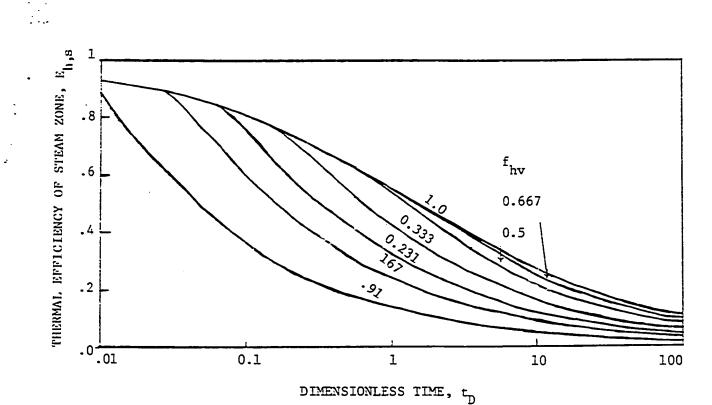


FIGURE G22: TEMPERATURE DISTRIBUTION WITH DISTANCE FOR RUN NO. 6



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FIGURE G23: FRACTION OF HEAT INJECTED IN STEAMFLOOD REMAINING IN STEAM ZONE. (f IS THE RATIO OF LATENT ENERGY TO TOTAL ENERGY INJECTED). (AFTER REF. 47)