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THE UNIVERSITY OF OKLAHOMA
GRADUATE COLLEGE

PRODUCTION PERFORMANCE SIMULATION OF GAS
AND GAS-CONDENSATE RESERVOIRS

A DISSERTATION
SUBMITTED TO THE GRADUATE FACULTY
in partial fulfillment of the requirements for the
degree of
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BY
FARIBORZ FARSHAD
Norman, Oklahoma
1975

PRODUCTION PERFORMANCE SIMULATION OF GAS
AND GAS-CONDENSATE RESERVOIRS

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PRODUCTION PERFORMANCE SIMULATION OF GAS AND GAS-CONDENSATE RESERVOIRS

CHAPTER I

INTRODUCTION AND PROBLEM FORMULATION

The earliest reported transportation and use of natural gas was by the Chinese. As early as 900 A.D., the Chinese transmitted natural gas in bamboo pipes, from coal beds to their salt workings, where brine was evaporated by heat from the burning gases.²³ The Chinese were centuries ahead of the peoples in the Western hemisphere in the use of natural gas for domestic and industrial purposes.

The earliest recorded use of natural gas in the U.S. was in the town of Fredonia, N.Y., in 1827. The gas was transported in lead pipes for the short-distance usage. However, it was not until 1947 when the change in the character of the gas industry occurred. Natural gas was transported from the Southwest to the East Coast through two converted liquid pipelines. Since then the consumption of natural gas has increased rapidly in the U.S. as it has in other parts of the world. By 1970, the high pressure gas transmission network was extended into all the lower 48

states; it included 269,610 miles of pipe and 4 million horsepower of compression.³³ The consumption of natural gas increased at an average rate of approximately 6.5 per cent per year and the percentage of the Nation's energy consumption it supplied rose from 13 per cent in 1945 to 33 per cent in 1970.

Natural gas produced from gas and gas condensate reservoirs provide a portion of the nation's future energy. Therefore, the prediction of the long term deliverability of such wells is of great importance. The prediction of the decline of average field pressure during the producing life of a field and accurate calculations of pressure losses in the producing system are essential in order that reservoirs may be produced efficiently and economically. In the completion and operation of a gas or gas condensate reservoir a given deliverability must be maintained with transfer at a certain pressure at the sales-point. The engineer is often faced with the decision as how to select and size the producing equipment necessary to maintain deliverability as the reservoir pressure declines. In the cases where the wells have already been completed, it may require larger tubing strings, enlarging flowlines or installation of compressors. In some cases, soon after completion of the well, it may be known that compression will eventually be necessary for depletion of the field. Obviously, the optimal combination of the various facilities and equipment is necessary in order to acquire the maximum economic return from the producing property.

A gas or a gas condensate production system is usually composed of five distinct elements. The five elements of the system can be characterized as follows:

1. Flow through the reservoir with the pressure drop from the average reservoir pressure to the bottom-hole flowing pressure.
2. Flow through the producing strings of the wells with the pressure drop from the bottom-hole flowing pressures to the wellhead flowing pressures.
3. Flow through the surface facilities including the gathering system and the processing, control and metering equipment with the pressure drop from the wellhead flowing pressures to the compressor station inlet pressures.
4. Compression in the station with the pressure increase from compressor station inlet pressure to compressor station discharge pressure.
5. Flow through the pipeline to the sales-point with the pressure drop from compressor station discharge pressure to delivery pressure.

In previous years, extensive work has been done on the flow behavior in the five elements of the system discussed above. However, there has been no major study on the integrated performance and deliverability of such a production system.

The objectives of this investigation are:

1. To develop models to investigate the overall performance of a dry gas and a gas-condensate production system.

2. To integrate all the components of the production system into a systematic delivery model to allow for the optimal selection of (a) producing tubing or casing strings, (b) gathering system, compressor station capacity and pipeline, (c) number of completions within the reservoir (spacing).
3. To predict the well deliverability, total field deliverability and long term deliverability to a fixed pressure point.
4. To analyze the effect of the condensate on the overall performance of the gas wells, a feature which is of great importance in the deliverability of the production system.
5. To analyze the important factors included such as (a) the combined behavior of the individual elements and the interplay between them, (b) to determine the minimum amount of data required to model the system adequately.

In this thesis, schemes for the solution of the above are presented. These schemes are incorporated into a computer model which solves the problem extremely efficiently.

CHAPTER II

MATHEMATICAL MODEL

In the previous chapter five basic components of a production system were introduced. In this chapter the basic equations governing flow behavior in the five basic components are presented.

Performance Prediction for a Dry Gas Reservoir

Back-Pressure Equation. - For many years, back-pressure tests on the gas wells have been used by the oil and gas industries to determine the capacity or the open-flow potential of a gas well. The first step toward calculating the open-flow potential or capacity was made in the classic paper by Rawlins and Schellhardt³¹ in 1937. They described and explained the back-pressure method of determining the capacity of a gas well to produce under various conditions of back pressure. They showed that a plot of volume-rate-of-flow versus the difference between the square of the static reservoir pressure and the square of the corresponding flowing bottom-hole pressure on log-log paper is a straight line. Figure 1 shows the back-pressure curve which is from the calculated values of P_{ws} and P_{wf} . The back pressure curve

is represented by an empirical equation; namely,

$$q = C(P_{ws}^2 - P_{wf}^2)^n \quad (2.1)$$

where

q = flow rate in Mscf/day

C = performance coefficient

P_{ws} = the average reservoir pressure in psia

P_{wf} = the flowing bottom hole pressure in psia

n = the reciprocal of the slope of the back-pressure curve.

From a theoretical standpoint, this equation can be considered as the steady state radial flow equation if $n = 1$, and can be stated as:

$$q = \frac{19.88 T_{sc} kh (P_{ws}^2 - P_{wf}^2)}{Z P_{sc} T_f^\mu g [\ln(r_e/r_w)]}$$

For a particular well, the terms k , h , T_f , T_{sc} , P_{sc} , Z , and $\ln(r_e/r_w)$ may be assumed constant which yields:

$$q = C(P_{ws}^2 - P_{wf}^2)$$

where

$$C = \frac{19.88 T_{sc} kh}{Z P_{sc} T_f^\mu \ln(r_e/r_w)}$$

This is strict Darcy flow. In addition, it assumes isothermal steady state flow and requires an average compressibility factor. For a particular well neither of these assumptions holds absolutely true and thus the exponent n must be introduced.

In the monograph 7, Rawlins and Schellhardt³¹ showed the relationship between absolute open flow and absolute formation pressure in the sand for different values of "n" ranging between $n = 0.1$ and $n = 20$. This is considerably beyond the practical range for n which is $0.5 \leq n \leq 1$. The variation of the exponent n between different wells has not been satisfactorily explained; however, many tests under actual field conditions and in the laboratory have shown the reliability of the back pressure equation. In low permeability reservoirs in which the wells do not stabilize quickly, it is often not possible to measure directly the stabilized performance without wasting large volumes of gas. For many tight reservoirs both economics and current focus on gas reserves preclude tests of sufficient duration to achieve stabilized flow for a sequence of flow rates. Riley³² introduced a stabilization factor (SF) for low permeability reservoirs, where because of a short period of testing, stabilization cannot be achieved. This factor applied to the short-term flow rate will give a reasonable approximation of the stabilized flow rate at the back-pressure used in the flow test.

Performance Prediction of Gas-Condensate Reservoir

As long as the reservoir fluid remains in single phase (gas) as the reservoir is depleted, the back-pressure equation presented previously is adequate for performance prediction of the well. However, most gas reservoirs

produce some hydrocarbon liquid, commonly called condensate, in the range of a few to a hundred or more bbls/day. This type of well suffers more rapid decline in productivity than that predicted by the theory for dry gas wells due to condensate accumulation within the reservoir and within the producing string. Because of high rates of liquid production, the well may not be able to produce against the required pipeline pressure which consequently necessitates compression. In other cases, the percentage of liquid at the bottom hole condition is sufficient that it loads the well and causes it to die. Figure 2 shows the percentage of liquid as a function of the reservoir pressure. This plot is the result of the laboratory analysis on the fluid sample obtained at the bottom-hole or the well-head. Most gas-condensates in fact behave in this manner, i.e. the condensate production increases as the reservoir pressure decreases. The condensate production is curtailed after a peak production is reached and the reservoir then behaves as a dry gas reservoir.

To take into account the above phenomena, the equation suggested by Fussel et al.¹⁸ was used to predict the performance of gas-condensate reservoirs. This equation can be written in the following form:

$$q = C \left(1 - \frac{V_o Z_g}{V_g Z_o} \right) (P_{ws}^2 - P_{wf}^2)^n \quad (2.4)$$

where V = volume percent of HCPV occupied by the gas or condensate phase. Figure 2 is a plot of V_o vs. pressure.

Flow in Wells

Single Phase (Gas) Flow in Wells. - A knowledge of the bottom-hole flowing pressure of a gas reservoir is of prime importance in predicting the reserves and deliverability of the gas in the reservoir. For this purpose, flowing sand-face pressure may be measured with a bottom-hole pressure gauge or computed from the well-head pressure. The pressure drop in the production string during gas flow is composed of two components:

1. The pressure drop across a static column of gas with the same average density as the flowing gas column.
2. Friction loss caused by flowing gas through the pipe.

Different methods for calculating and combining the values for the two components have been suggested.^{13,23,27,35} The equations commonly used in practice for calculating the flow of gas in a gas well are based on the following assumption:

- A. The kinetic energy change is negligible.
- B. The flow is isothermal.
- C. There is no work done by the gas in flow.

The most realistic equation which has received the most widespread use for calculating the pressure drop in the flowing gas well is that of Smith.³⁵ Smith gave the equation for

calculating pressure in a dry gas well (neglecting kinetic energy change, and assuming that temperature and compressibility are constant at their average value) as:

$$q = \left[\frac{c_f d^5 (p_{wf}^2 - e^s p_{tf}^2)}{f T_{avg}^2 z_{avg}^2 (e^s - 1)} \right]^{0.5} \quad (2.5)$$

Rearranging would yield:

$$p_{wf} = \left[p_{tf}^2 + \frac{(f q^2 T_{avg}^2 z_{avg}^2)(e^s - 1)}{c_f d^5 e^s} \right]^{1/2} e^{s/2} \quad (2.6)$$

$$s = \frac{0.0375 \text{ GD}}{T_{avg} z_{avg}}$$

Multiphase Flow (Gas and Condensate in Wells). - In the petroleum industry the production of gas-condensate through wells involves the flow of mixed fluid phases. In this case, natural gas with light liquid hydrocarbons are produced simultaneously and the flow mixture is two-phase. Several methods^{1,3,4,9,15,17,22,29,30} have been developed to compute and predict the flow patterns, liquid hold-up and pressure losses occurring during two-phase flow through vertical pipe. These pressure-loss prediction methods are made up of a combination of pressure-loss correlations and various fluid physical property correlations.

The general pressure gradient equation³⁰ for a vertical flow can be written as:

$$\frac{dP}{dz} = \left(\frac{dP}{dz} \right)_{elev} + \left(\frac{dP}{dz} \right)_{fric} + \left(\frac{dP}{dz} \right)_{acc} \quad (2.7)$$

where:

$\left(\frac{dP}{dz} \right)_{elev}$ = pressure drop caused by elevation change,

$\left(\frac{dP}{dz} \right)_{fric}$ = pressure drop caused by friction,

$\left(\frac{dP}{dz} \right)_{acc}$ = pressure drop caused by acceleration.

The $(dP/dz)_{elev}$ depends on the density of the two-phase mixture and is usually calculated by a liquid holdup value. The $(dP/dz)_{fric}$ is due to the friction losses and a two phase friction factor must be evaluated. The $(dP/dz)_{acc}$ depends on the flow velocity and is considered negligible except in the cases of high flow velocity. Except for the latter case, most of the pressure drop is attributed to elevation. These components can be evaluated by many different types of correlations. Beggs and Brill⁶ classified the correlations according to their complexity and the methods used to evaluate each of the components as follows:

1. No Slip, No Flow regime:^{3,17,30} liquid holdup is not considered in the computation of the density. The liquid holdup and the friction losses are computed empirically. No distinctions are made among flow regimes.

2. Slip considered, No Flow regime:²² liquid holdup is considered in the computation of the density. The friction losses are based on the composite properties of the liquid and gas. No distinctions are made among flow regimes.
3. Slip considered, Flow regime considered:^{1,4,9,15,29} The calculated density term considers liquid holdup. Liquid holdup is determined from the difference between gas and liquid velocities (slip velocity). The friction losses are determined from the fluid properties of the continuous phase. Distinct flow regimes are considered.

Orkiszewski²⁹ presented a method in which the two-phase pressure drops can be accurately predicted over a wide range of well conditions and it considered the slip and the flow regime. Orkiszewski's work is a composite of several methods tabulated below:

<u>METHOD</u>	<u>FLOW REGIME</u>
Griffith ¹⁹	Bubble
Griffith and Wallis ²⁰	Slug (density term)
Orkiszewski ²⁹	Slug (friction gradient term)
Duns and Ros ¹⁵	Transition
Duns and Ros ¹⁵	Mist

Figure 3 shows the flow regimes of two-phase flow considered by Orkiszewski²⁹ and can be briefly described as follows:

1. Bubble Flow (Fig. 3-A)

In this regime the pipe is almost completely filled with the liquid phase and the free gas phase is small. Free gas is present as small bubbles and the liquid is the continuous phase.

2. Slug Flow (Fig. 3-B)

In this flow pattern more gas bubbles coalesce to form larger, bullet shaped bubbles. These bubbles are separated by a slug of liquid and are surrounded by a thin liquid film.

3. Transition Flow (Fig. 3-C)

In this regime the change from a continuous liquid phase to a continuous gas phase occurs. Although the liquid effects are significant, the gas phase effects are predominant.

4. Mist Flow (Fig. 3-D)

In this regime the liquid droplets are carried in the continuous gas phase. The pipe wall is coated with a liquid film and the gas phase predominantly controls the pressure gradient.

In two phase flow, both the friction loss gradient and the fluid density are influenced by the flow regime type, and all three terms (equation 2.7) are a function of temperature and pressure. Therefore, to use equation 2.7 the following procedure must be followed:

1. Increment the flow string so that the fluid properties do not markedly change within any of the increments.
2. For each increment, determine the flow regime and compute the friction loss and average fluid density.
3. Each increment must be evaluated by using an iterative method.

Equation 2.7 can be written in the following form:

$$\Delta P = \left[\frac{1}{144} \frac{\bar{\rho} + \tau_f}{1 - w_t q_g / 4637 A_p^2 \bar{P}} \right]_k \Delta Z_k \quad (2.8)$$

Determination of Flow Regime

Griffith and Wallis²⁰ defined the boundary between bubble and slug flow and Duns and Ros¹⁵ defined the boundaries for the slug, transition and the mist flow. The flow regimes are determined by testing whether the variable q_g/q_T or \bar{V}_g or both fall within the limits tabulated below:

<u>Limits</u>	<u>Flow Regime</u>
$q_g/q_T < (L)_B$	Bubble
$q_g/q_T > (L)_B, \bar{V}_g < (L)_S$	Slug
$(L)_M > \bar{V}_g > (L)_S$	Transition
$\bar{V}_g > (L)_M$	Mist

where

$$\bar{V}_g = q_g (\sqrt{\rho_g / g \sigma}) / A_p \quad (2.9)$$

$$(L)_B = 1.071 - (0.2218 V_t^2 / D_h), \text{ with limit} \quad (2.10)$$

$$(L)_B \geq 0.13$$

$$(L)_s = 50 + 36 \bar{V}_g q_\ell / q_g \quad (2.11)$$

$$(L)_M = 75 + 84 (\bar{V}_g q_\ell / q_g) \quad (2.12)$$

\bar{V}_g = dimensionless gas velocity

Gathering System

Flow of gas through gas gathering system. - The design of pipelines for gathering natural gas in the oil field has been extensively studied and a number of equations have been used to predict the pressure drop in the gas gathering system.^{5,7,28,36}

The Weymouth³⁸ formula is most commonly used for this purpose within the oil and gas industry and has been found to be adequate. The Weymouth formula, in its most common form, may be expressed as:

$$q = 433.45 \frac{T_s}{P_s} d^{2.667} \left(\frac{P_1^2 - P_2^2}{GT_a LZ_a} \right)^{0.5} \quad (2.13)$$

Flow of two-phase in the gathering system. - Two-phase flow has been of considerable interest in the petroleum industry due to the desirability of accurately calculating the pressure losses that occur in the gathering system.

Gas-liquid mixtures have been transported over relatively long distances in a common line due to centralized gathering systems, particularly in offshore operations. Large pressure drops occur in long two-phase flow lines. Pressure losses in two-phase gas-liquid flow vary considerably from those encountered in single-phase flow; in most cases, an interface exists and the gas slips past the liquid. Several published methods exist for prediction of pressure drop in two-phase flow through horizontal pipe.^{4,14,21,25,39} Many of the correlations use a holdup value in calculating the density term used in the friction and acceleration pressure drop components.

Eaton¹⁶ developed correlations for liquid holdup and the two-phase friction factor. Equation 2.7 can be written for multiphase horizontal flow in the following form:

$$\frac{dP}{dL} = \left(\frac{dP}{dL} \right)_f + \left(\frac{dP}{dL} \right)_{acc} \quad (2.14)$$

where Eaton's¹⁶ friction factor is:

$$\left(\frac{dP}{dL} \right)_f = \frac{f_{\rho_n} V_{mix}^2}{2g_c d} = \frac{f W_{mix}^2}{2g_c d A_p^2 \rho_n} \quad (2.15)$$

and Eaton's¹⁶ acceleration term is:

$$\left(\frac{dP}{dL} \right)_{acc} = \frac{W_{liq} \Delta V_{liq}^2 + W_{gas} \Delta V_{gas}^2}{2g_c q_{mix} dx} \quad (2.16)$$

Note that in equation 2.14 the term for potential energy or elevation change is zero.

Compression

Compression is an important element of the production system, and the station inlet pressure at a given deliverability determines the remaining parameters for the reservoir and the production system. The suction pressure of the compressor reflects back to the producing string pressure at the wellhead. The producing string pressure influences the back pressure on the sandface which directly influences feed-in rate.

The engineer in the field is frequently required to determine the approximate horsepower required to handle a certain volume of gas at some intake conditions to a given discharge pressure. Therefore, the selection of proper compressor size is of great importance. Many compression requirements involve conditions beyond the practical capability of a single compression stage. Too great a compression ratio (absolute discharge pressure divided by absolute suction pressure) causes excessive discharge temperature and other design problems. It, therefore, may become necessary to combine elements or groups of elements in series to form a multistage unit, in which there will be two or more stages of compression. The gas is frequently cooled between stages to reduce the temperature and volume entering the following stage. Each stage involves an individual compression. It is sized to operate in series with one or more additional compressors and even though they may all operate from one power source, each is still separate.

The behavior of the compressor station can be represented by two methods: Isothermal Compression and Adiabatic (Isentropic) Compression. Isothermal compression occurs when the temperature is kept constant as the pressure increases. This requires continuous removal of the heat of compression. Adiabatic (isentropic) compression is obtained when there is no heat added to or removed from the gas during compression. Adiabatic compression calculations give the maximum theoretical work or horsepower necessary to operate the compressor between any two pressure limits, whereas isothermal compression calculations give the minimum theoretical work or horsepower necessary to compress a gas.

In this study, the equation of adiabatic theoretical horsepower given by Katz et al.²³ was used. This equation, which is to find the adiabatic theoretical horsepower to compress 1 mmcf/day at 60°F. and 14.65 psia, is written as follows:

$$\text{ATHP} = 0.08531 \frac{k}{k - 1} T_{\text{suc}} \left[r^{\frac{Z_{\text{suc}}(k-1)}{k}} - 1 \right] \quad (2.17)$$

where:

$$r = P_{\text{dis}} / P_{\text{suc}}$$

P_{dis} = discharge pressure of gas, psia

P_{suc} = suction pressure of gas, psia

$k = c_p / c_v$ = ideal-gas specific-heat ratio

The total brake horsepower (BHP) required is given by

$$BHP = \left(\frac{ATHP}{E} \right) q_{sc} \left(\frac{14.73}{14.65} \right) \quad (2.18)$$

where

E = overall efficiency

Flow of Gas Through Pipeline

The Panhandle A⁸ Formula. - As previously discussed in the gas gathering section of this chapter, the Weymouth³⁸ formula is suitable for use in the design of gathering system and short pipelines. For larger diameter long lines designed for transmission and gas delivery, the panhandle is used. The formula can be written in the following form:

$$Q = 435.87 E \left(\frac{T_s}{P_s} \right)^{1.0788} \left(\frac{P_1^2 - P_2^2}{T_f L Z_a} \right)^{0.5394} \left(\frac{1}{G} \right)^{0.4606} d^{2.6182} \quad (2.19)$$

where E = efficiency factor.

CHAPTER III

TOTAL FIELD DELIVERABILITY

The total field or reservoir back-pressure equation can be obtained by merely summing the performance coefficient (C) for all the wells in the reservoir. This can be shown as follows:

$$\text{Reservoir Deliverability} = q_{\text{reservoir}} = \left(\sum_{j=1}^{N_w} C_j \right) (P_{ws}^2 - P_{wf}^2)^n$$

where

C_j = performance coefficient of well j

N_w = number of wells

The above equation can be written on an average per well basis, which is:

$$(q_{sc})_{\text{avg}} = C_{\text{avg}} (P_{ws}^2 - P_{wf}^2)^n$$

Note that the value of " n ", the exponent of the back-pressure equation, is assumed the same for all wells. In most cases this is a very reasonable assumption and the value of " n " observed in practice turns out to be very nearly constant in many individual reservoirs. In fact, it is often adequate for field calculations to consider " n " to be constant on a regional basis.

CHAPTER IV

PREDICTION OF RESERVOIR PERFORMANCE

In order to forecast the deliverability of a gas and gas-condensate production system, the system deliverability must be coupled with a material balance equation for the reservoir. The following equations are obtained on the basis of a material balance technique¹² for reservoirs in which there is no water influx or water production.

Gas initially in reservoir = gas produced

+ gas remaining

thus,

$$G = G_p + (G - G_p) \quad (3.1)$$

If PV = 379 GZRT, then

$$\frac{P_i V_i}{379 Z_i RT} = G_p + \frac{PV_i}{379 ZRT} \quad (3.2)$$

and

$$\frac{P}{Z} = \frac{P_i}{Z_i} - \frac{379 RT}{V_i} G_p \quad (3.3)$$

Solving for G_p would yield

$$G_p = \frac{V_i}{379 RT} \left(\frac{P_i}{Z_i} - \frac{P}{Z} \right) \quad (3.4)$$

or

$$G_p = G \frac{Z_i}{P_i} \left(\frac{P_i}{Z_i} - \frac{P}{Z} \right) \quad (3.5)$$

Then

$$G_p = G \left(1 - \frac{P/Z}{P_i/Z_i} \right) \quad (3.6)$$

or

$$G_p = G \left(\frac{\frac{P_i/Z_i - P/Z}{P_i/Z_i}}{1} \right) \quad (3.7)$$

Equation 3.7 is used to calculate the cumulative gas production during any period of production. Equation 3.7 can also be written in terms of the recovery factor (RF):

$$G_p = G \times RF$$

where the recovery factor is defined as follows:

$$RF = \frac{P_i/Z_i - P/Z}{P_i/Z_i} \quad (3.8)$$

At abandonment conditions the recovery factor is given as follows:

$$RF = \frac{P_i/Z_i - P_a/Z_a}{P_i/Z_i}$$

P_a/Z_a corresponds to the minimum rate at which gas production will no longer be economical (abandonment rate).

Decline rate production. - In the prediction of the long term reservoir deliverability, we must estimate the start

of decline in the production rate. This can be simply computed by the following relation:

$$\text{Time to decline} = \frac{\text{Cumulative gas production}}{\text{Constant rate production}} \quad (3.9)$$

The important point is that the time to decline results from the joint solution of the material balance equation for the reservoir and the overall system flow equations.

CHAPTER V

SOLUTION TECHNIQUES

A constant volume gas expansion type of reservoir is considered. There is no water production nor water encroachment. The objectives of this study are to determine:

- I. The maximum deliverability of a production system
- II. Based on the predicted deliverability of the system:
 1. the life of the well
 2. the cumulative gas production as a function of time.
 3. the average reservoir pressure as a function of time.
 4. the average delivery rate as a function of time
- III. The effect on the above with the change of the following parameters:
 1. size of the producing string
 2. size of the compressor
 3. number of the wells
 4. flow rate
- IV. Repeat the above procedure for a gas-condensate system.
- V. Compare the results of the production performance for the dry gas reservoir with the production performance of the gas-condensate reservoir.

The equations used to describe the behavior of the various components to calculate the deliverability of the gas and the gas-condensate production system were the following:

I. Reservoir

1. Dry Gas: back-pressure equation
2. Gas-Condensate: Fussel et al. equation

II. Production String

1. Dry Gas: Smith equation
2. Gas-Condensate: Orkiszewski correlation

III. Gathering System

1. Dry Gas: Weymouth equation
2. Gas-Condensate: Eaton correlation

IV. Compressors: Adiabatic compression equation**V. Pipeline: Panhandle A formula**

The calculation of the maximum deliverability for a production system involves a trial and error solution. A flow rate, Q , is assumed and the flowing bottom-hole pressure will be calculated in two ways: (1) from the pipeline side, and (2) from the reservoir side. The calculation of the bottom-hole pressure from the line side is as follows:

1. With the assumed value of Q , and the designated delivery pressure, use the panhandle formula to compute the inlet pressure of the pipeline (this inlet pressure calculated is the P_{dis} of the compressor).
2. With the discharge pressure computed in step 1, use the adiabatic compression equation to evaluate the suction pressure at the compressor (this calculated P_{suc} is the outlet pressure of the gathering system).

3. With the gathering system outlet pressure computed in step 2, use the Weymouth equation (Eaton correlation for gas-condensate systems) to calculate the inlet pressure of the gathering system (the inlet pressure calculated is the wellhead flowing pressure).
4. With the wellhead flowing pressure computed in step 3, use the Smith equation (Orkiszewski correlation for gas-condensate systems) to calculate the bottom-hole flowing pressure.

The calculation of the bottom-hole pressure from the reservoir side is as follows:

With the assumed value of Q coupled with other reservoir properties, use the back-pressure equation (Fussel et al. equation for gas-condensate systems) to calculate the bottom-hole flowing pressure.

If the computed pressure from the line side is greater than the computed pressure from the reservoir side, the value of Q is too high and the above procedure must be repeated with a smaller Q. The iterative process is continued until a value of Q is obtained for which the bottom-hole pressures computed from both sides are in close agreement. The method of false position was used to accelerate the iterative procedure.

If there is more than one completion in the reservoir, it is assumed that all wells are identical in every respect, and that the production rates are the same.

Production Performance Under a Prescribed
Production Rate or Capacity
Production Rate

If the maximum delivery rate computed is higher than the prescribed flow rate, it indicates that the production system can meet the specified requirements. Thus for a given system, the time intervals over which the requirements can be met and those over which they cannot must be identified. In the former, the performance under the specified schedule is determined and in the latter the performance under capacity production is determined (capacity production is a steady decline in deliverability).

Equation 3. is used to calculate the time-to-decline under the specified shcedule. The performance under capacity production at any time step J can be computed as follows:

1. Specify the system deliverability, Q_J .
2. Calculate the average reservoir pressure required to maintain the production rate Q_J .
3. Calculate the average system deliverability $(Q_{avg})_J$:

$$(Q_{avg})_J = \sqrt{Q_J Q_{J-1}}$$

4. Calculate the cumulative production, $(G_p)_J$, at the end of time step J as follows:

$$(G_p)_J = G \left[\frac{\left(\frac{P_{res}}{Z} \right)_i - \left(\frac{P_{res}}{Z} \right)_J}{\left(\frac{P_{res}}{Z} \right)_i} \right]$$

5. Calculate the incremental gas production:

$$(\Delta G_p)_J = (G_p)_J - (G_p)_{J-1}$$

6. Calculate the incremental time:

$$\Delta t_J = \frac{(\Delta G_p)_J}{(Q_{avg})_J (365)}$$

7. Calculate cumulative time at the end of time step J:

$$(t_{cum})_J = t_{J-1} + \Delta t_J$$

The simulation of the production performance and deliverability of gas and gas-condensate reservoirs was carried out on a digital computer, since performing such a heavy load of mathematical operations either by hand or on a desk calculator is not feasible. The computer model is written in FORTRAN IV language for the IBM system/370. The FORTRAN IV language is especially useful in writing programs for applications that involve mathematical computation and other manipulations of numerical data.

The computer model was made completely flexible. Any type of production system can be incorporated, and any number of wells can be investigated within the same reservoir. The model can be run without a portion or portions of the production system such as pipeline, compressor and gathering system.

Thus, fifteen subroutines were prepared, each performing a specific task. Subroutine QCALC coupled with BHP were designed to calculate the optimum deliverability of the production system. Following is the list of the remaining subroutines and their functions:

FRIFAC	Friction factor calculation (Colebrook and White ¹¹)
GASVIS	Gas viscosity calculation (Lee <u>et al.</u> ²⁴ correlation)
MUFF	Two-phase flow in vertical pipe (Orkiszewski ²⁹ correlation)
MUFFIH	Two-phase flow in horizontal pipe (Eaton <u>et al.</u> ¹⁶ correlation)
OWLVIS	Live oil viscosity calculation (Chew and Connally ¹⁰)
OILRAT	Condensate production rate (see Appendix D)
RESP	Reservoir pressure calculation
SOLZ	Iterative method of linear inverse interpolation (method of false position)
SURFAC	Live oil surface tension (gas-oil system) (Baker and Swerdloff ²)
ZFAC0 ZFAC1 ZFAC2}	Compressibility factor (Z) calculation (Sarem ³⁴)

CHAPTER VI

RESULTS AND DISCUSSION

This investigation involved a novel technique which determined the maximum production capacity and predicted the long-term deliverability of the integrated production system for a gas and a gas-condensate reservoir. The solution method and the organization of the computer models were presented in Chapters II through V. The data used to obtain the tables and the curves, referred to in this discussion, are presented in Table 1 of Appendix A. The method of investigation involved the interpretation of a series of runs which were made for all cases listed in Table 2, Appendix A.

The size of the producing tubing strings, the compressor station capacity, the number of well in the reservoir and the flow rate were varied in a series of simulation runs and the system production performance were obtained.

Simulation Results of the Dry Gas Model

The results of the simulation study are shown in Tables 3 and 4 and Figures 4 through 19 in Appendix B and C, respectively.

Table 3 indicates the flow behavior of a reservoir with 10 wells producing through (1) 2-1/2-inch tubing and (2) 7-inch casing with station capacities of 3325, 6650, and 8300 HP. Note that the magnitude of the total field deliverability and compressor discharge pressure of the systems increase, and the bottom-hole flowing pressure, the well-head flowing pressure and the compressor station suction pressure decrease as the size of the production string and the compressor station capacity increase.

Similarly, Table 4 shows the flow behavior of a reservoir with (1) 10 wells, (2) 20 wells, and (3) 30 wells, producing through 2-1/2-inch tubing at a station capacity of 8300 HP. The comparison of these three systems shows that the total field deliverability, the bottom-hole flowing pressure, the well-head flowing pressure, the compressor station suction pressure and the compressor station discharge pressure increase as the number of completions increases (well spacing decreases). The system deliverability increased by 41.8% when the number of wells increased from 10 to 20; however, it only increased by 8.59% when the number of wells increased from 20 to 30.

Figures 4 through 7 of Appendix C show the total field deliverability vs time for 10 wells producing through 2-1/2-inch tubing and 7-inch casing at station capacity of 6650 HP. Flow rates of 50, 100, 150 and 200 MMscf/day are prescribed. The deliverabilities of the wells are represented by their respective deliverability curves. The comparison

of these figures indicates that the maximum deliverability for 2-1/2-inch completions and 7-inch completions are 134.82 and more than 200 MMscf/day respectively. The increase in total field deliverability is due to the reduced friction losses resulting from the larger area available for flow in the producing string.

Figure 4 shows that in the 2-1/2-inch completions the total field deliverability declines below 50 MMscf/day in the third quarter of the ninth year while in 7-inch completions the total field deliverability does not decline until the last quarter of the thirteenth year. However, the latter reaches the abandonment rate over a year earlier. Figure 5 shows similar behavior except the decline production is reached earlier because of the twofold increase in the production rate.

Figures 6 and 7 indicate that the 2-1/2-inch completions are performing under capacity production and that the total field deliverability is declining in spite of the use of compression throughout its productive life.

Figures 8-11 illustrate the behavior of a reservoir with 10 wells producing through 2-1/2-inch tubing and 7-inch casing at 3325, 6650, and 8300 HP capacities. Prescribed flow rates of 100 and 200 MMscf/day are shown. Figures 8 and 10 indicate that the total field deliverability of the production systems (2-1/2-inch and 7-inch completions) with 3325 HP station capacity would decline earlier than the other systems with larger compressor size. The production

system with larger compressor station capacity reaches the abandonment rate earlier than the other systems. Figures 9 and 11 exhibit the performance of the systems described above under capacity production.

Figure 12 shows that the total field deliverability is increased as the larger production string is installed. It is further increased as the size of the compressor is increased.

Figures 13-15 illustrate the effect of the compressor horsepower on the total field deliverability. The curves show the behavior of the production system (2-1/2-inch tubing and 7-inch casing) as the compressor station capacity is increased. In Figure 13 the curve indicates that as the compressor station capacity increases the deliverability of the system increases rapidly. However, the rate of increase in the deliverability decreases when a compressor station capacity of 4000 HP is reached. Furthermore, there will be a point in the curve where the slope of the line will be zero which illustrates that there will be no more increase in the deliverability of the system in spite of the use of a larger compressor.

Figure 16 shows the behavior of a reservoir with 10 wells producing through 2-1/2-inch tubing with station capacity of 3325 and 6650 HP; 10 wells producing through 7-inch casing and 6650 HP station capacity; and 20 wells producing through 2-1/2-inch tubing and 6650 HP capacity. These curves

illustrate the effects of the size of the compressor, the size of the production string and the number of completions.

Figure 17 is a graph of the cumulative gas production versus time for 2-1/2-inch tubing and 7-inch casing completions. This curve indicates that the 7-inch casing completions have higher recovery factors than 2-1/2-inch completions. However, the abandonment rate is reached 1.75 years later in the latter case.

Figure 18 shows the overall system deliverability for the cases discussed in Figure 16. This curve shows the deliverability of the system versus the average reservoir pressure. This curve illustrates the maximum deliverability of the system as a function of the average reservoir pressure or gas in place. This curve is of prime importance because it exhibits the deliverability under the influence of all the components of the system.

Figure 19 shows the total field deliverability as a function of the compressor station suction pressure for 10 wells producing through 2-1/2-inch tubing at 6650 and 8300 HP capacity. This curve is also of prime importance since the suction pressure of the compressor will reflect back to the well-head flowing pressure, which in turn will reflect back to the bottom-hole flowing pressure and consequently affect the feed-in rate.

Simulation Results of the Gas-Condensate Model
and Comparison of the Production Performance
of the Gas and Gas-Condensate Reservoirs

The effect of the compressor size on the flow behavior of the production system was first studied for 10 wells producing through 2-1/2 inch tubing at 3325, 6650 and 8300 HP capacity (see Table 5, Appendix B). Second, the effect of the number of completions on the flow behavior and system deliverability was investigated for 10, 20, and 30 wells producing through 2-1/2 inch tubing at 6650 HP compressor station capacity. Third, the effect of the size of the production string on the total field deliverability was studied for 10 wells producing through 2-1/2 inch tubing and 7 inch casing at 6650 HP station capacity.

Table 5 shows that the maximum total field deliverabilities increased as the compressor size increased. The maximum deliverability of the system with 3325 HP compressor was predicted to be 23.6 MMscf/day and 192 bbls/day condensate while the maximum deliverability of the system with 8300 HP compressor is computed to be 35.6 MMscf/day and 290.9 bbls/day condensate. In these runs the assumption is made that the gas and the condensate are separated by the separator at the well head, and that there is no pressure loss through the separator. However, in actual field operation there is a pressure drop through the separator with the pressure drop being in the range of a few psia. Table 5 also shows the pressure drop through the production string.

This significant pressure drop is due to the multiphase flow through the pipe.

Table 6 illustrates the effect of an increase in the number of completions in the reservoir. It indicates that as the number of completions in the reservoir increases the gas deliverability and the condensate production of the system increases.

Figure 20 shows the effect of increasing the diameter of the production string on the total field deliverability. It indicates that as the size of the production string increases, the total field deliverability increases.

Figure 21 compares the total field deliverability for 10 wells producing through 2-1/2 inch tubing at 3325 HP and 8300 HP station capacity; 20 wells producing through 2-1/2 inch tubing at 8300 HP capacity; and 10 wells producing through 7-inch casing at 8300 HP capacity. In all the systems investigated (Tables 5 and 6 and Figures 20 and 21) and discussed above, the prescribed flow rate was 50 MMscf/day of gas and 400 bbls/day of condensate. It is apparent that the reservoir and the production system described will not produce under the prescribed production rates.

Comparison of the Production Performance of the Gas and Gas-Condensate Reservoirs

The results of the simulation studies and their comparison are illustrated in Tables 7 and 8 (Appendix B) and Figures 22-28 (Appendix C). The comparison of the results

from simulation of the dry gas and the gas-condensate models shows some noticeable differences.

Table 7 shows the fluid flow behavior for 10 wells producing through 2-1/2 inch tubing and 7-inch casing with compressor capacity of 3325 HP. The prescribed flow rate is 50 MMscf/day of gas for the dry gas case and 50 MMscf/day of gas and 400 bbls/day condensate for the gas-condensate system. This table shows the difference in the behavior of the reservoirs and the effect of each element of the system on the overall production performance. For the case of 2-1/2 inch completions the maximum total field deliverability of a dry gas system is decreased from 118 MMscf/day to 23.6 MMscf/day. Similarly, in the case of the 7-inch completions the total field deliverability is decreased from 164.7 MMscf/day to 24 MMscf/day. Also, note that there is no marked difference in the overall performance of the 2-1/2 inch completions and the 7-inch completions of the gas-condensate system. The significant reduction in the total field deliverability and large increase in the pressure losses through the producing equipment is primarily the result of multiphase flow phenomena in the producing string and not deterioration of reservoir deliverability.

Table 8 shows the comparison of the 7-inch completions in the dry gas and the gas-condensate reservoir with the compressor station capacity of 8300 HP.

Figures 22 through 25 show the comparison of the performance of the dry gas and the gas-condensate reservoirs

for 10 wells producing through 2-1/2 inch tubing and 7-inch casing at a station capacity of 6650 HP. The prescribed gas flow rates are 50 and 100 MMscf/day and 400 bbls/day of condensate.

In figure 22, in the case of the dry gas, the total field deliverability declines below 50 MMscf/day (prescribed flow rate) in the last quarter of the ninth year. However, in the case of the gas-condensate, the flow system cannot sustain a production rate of 50 MMscf/day initially and its entire production life is characterized by capacity decline. The abandonment rate is reached six years earlier in the gas-condensate reservoir.

Figure 23 indicates that for the 7-inch completions in the gas-condensate reservoir, the total field deliverability declines below 50 MMscf/day in the first quarter of the first year and that total field deliverability of the dry gas system does not decline until the thirteenth year of production is reached.

Figures 24 and 25 show the comparison of the same systems with the prescribed flow rate of 100 MMscf/day.

Figure 26 shows the effect of the size of the production string on the total field deliverability of both dry gas and gas-condensate systems. It indicates that the size of the producing string has greater influence on the overall behavior of the dry gas system than that of the gas-condensate system.

Figures 28 and 29 show the effect of the compressor station capacity on the total field deliverability of the dry gas and gas-condensate systems.

CHAPTER VII

SUMMARY AND CONCLUSIONS

A mathematical simulation procedure for computing the dry gas and the gas-condensate field deliverability to a set delivery pressure at some terminal point has been developed.

The features of these procedures can be summarized as follows: (1) They provide simulations of the existing field producing system and prediction of the future performance. (2) The models allow the study of a system composed of several wells within a reservoir. (3) Models developed show the significance of the individual elements of the production system and the interplay between them. (4) Models show the significance of the flow behavior in the technology of the production, especially two-phase flow.

Conclusions from the results presented are:

1. The system deliverability could be increased by analyzing all parts of the system simultaneously in an integrated manner. In some cases, the situation may dictate the use of larger tubing strings and gathering flow lines. In other cases, a larger compressor may be desired. The method presented allows one to maximize deliverability by the design of the production strings, gathering system, compressor station

capacity and pipeline. The optimal design can be determined by computing the effects of the parameters, size of tubular goods and production strings, on reservoir behavior.

2. The scheme allows optimal gas well spacing by the selection of the number of wells within the field required to meet given production targets.

3. Comparisons made between the dry gas flow and two-phase flow production systems show that the deliverability of the gas-condensate, compared to that of a dry gas, is greatly reduced. This is due to condensate accumulation in the region of the producing wells and the pressure losses through the producing strings and gathering system attributed to multiphase flow.

4. The techniques described here provide important information such as (a) maximum well deliverability, (b) total field deliverability, (c) overall system deliverability, and (d) the production life of the reservoir. This information allows the selection of appropriate producing strings and tubular goods which will achieve a maximum cash flow or maximize some similar type criterion over the life of the field.

5. The simulation studies in this report will not account for erratic and unpredictable production problems, such as scale, corrosion, equipment failure and other problems which may become more serious as abandonment conditions are approached.

6. Methods using the simulation of mathematical models are very powerful and useful and could be used to predict the system performance of a reservoir in an economical amount of computer time.

CHAPTER VIII

RECOMMENDATIONS

Further considerations that must be taken into account before determining the optimum sizing of the production system and optimal well spacing include: the availability of equipment, operating cost and agreements, governmental rules and regulations, cost of equipment and many other limitations.

Therefore, the design of the project must be fully evaluated and the results of such an engineering study should be combined with (1) reservoir production optimization solved efficiently by using a dynamic programming approach and (2) economic analysis of the project to arrive at the most profitable operation.

NOMENCLATURE

A_p	area of pipe, sq. ft.
ATHP	adiabatic theoretical horsepower, HP
B_g	gas formation volume factor, bbl/SCF
C	performance coefficient
C_f	constant, 1.5×10^3
d	inside diameter of production string, in.
D	formation depth, ft.
D_h	hydraulic pipe diameter ($4 \times A_p$ /wetted diameter), ft.
E	efficiency
f	friction factor, dim.
g	acceleration of gravity, ft/sec^2
g_c	gravitational constant, $\text{ft-lb(mass)}/\text{lb(force)} \cdot \text{sec}^2$
G	gas gravity (air = 1)
h	formation thickness, ft.
HCPV	hydrocarbon pore volume, ft.^3
k	formation permeability, md
L	length of the pipe, ft.
$(L)_B$	bubble-slug boundary, dim.
$(L)_M$	transition-mist boundary, dim.
$(L)_S$	slug-transition boundary, dim.
n	the reciprocal of the slope of the back-pressure curve

N_w	number of wells, dim.
P	pressure, psi
\bar{P}	average pressure, psia
ΔP	pressure drop, psi
P_{tf}	flowing tubing or casinghead pressure, psi
P_{wf}	flowing bottomhole pressure, psi
P_{ws}	average reservoir pressure, psi
q	volumetric flow rate, cu. ft./sec
Q	volumetric flow rate, cu. ft./day
r	compression ratio
r_e	outer radius, ft.
r_w	well bore radius, ft.
R	universal gas constant (per mole), $\text{psi}\cdot\text{ft}^3/(1\text{b. mole R}^\circ)$
T	temperature, $^\circ\text{R}$
Δt	time increment, year
V	fluid velocity, ft./sec
ΔV_g^2	$V_g^2(P_1, T_1) - V_g^2(P_2, T_2)$
ΔV_L^2	$V_L^2(P_1, T_1) - V_L^2(P_2, T_2)$
W	mass flow rate, lb/sec
Z	compressibility factor
Z	depth from wellhead, ft.
ΔZ	increment of depth, ft.
μ	viscosity, cp
$\bar{\rho}$	average flowing density, lb/cu.ft.
ρ_n	two phase density, lb/cu.ft.
σ	surface tension, lb/sec^2

τ_f friction loss gradient, psf/ft
 \bar{V} $q_g (\sqrt{\rho_g / g \sigma})$, dimensionless gas velocity

Subscripts

a average, abandonment

cum cumulative

dis discharge

f flowing, friction

g gas

i initial

J time step

k increment

l liquid

Liq liquid

Mix mixture

o oil

p production

res reservoir

s, sc standard condition

suc suction

T total

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APPENDIX A

TABLE 1
DATA USED IN THIS STUDY

Gas Properties

$\gamma_g = .64$, gas sp. gravity
 $\mu_g = 0.0175$ cp, gas viscosity

Condensate Properties

$\gamma_o = 0.8$, condensate sp. gravity
 $\mu_o = 1.419$ cp, condensate viscosity
 $\sigma_o = 32$ dynes/cm, condensate surface tension
 $\rho_o = 42.45$ lbm/ft³, condensate density
 $B_o = 1.419$ bbls/stb, condensate formation volume factor

Reservoir Data

$N_w = 10$, number of wells (well spacing)
 $C_{avg} = 0.0742$ (MMscf/day)/psia^{2/n}, performance coefficient
 $n = 0.747$, exponent of the back pressure equation
 $P = 1500$ psia, average reservoir pressure
 $T = 120^{\circ}\text{F}, 580^{\circ}\text{R}$, reservoir temperature
 $G = 250,000$ MMscf, gas in place

Producing Tubing Strings Data

$D = 5000$ ft, well depth
 $d = 2\frac{1}{2}$ inch, size of the tubing
 $T_{avg} = 95^{\circ}\text{F} = 555^{\circ}\text{R}$, average flowing temperature
 $Z_{avg} = 0.95$, average compressibility factor

Surface Facilities Data

$C_3 = 7.63 \times 10^5$ (scf/day)/psia², the average flow conductivity of the gathering system

TABLE 1 (Continued)

Compressor Station Data

BHP = 6650 HP, compressor station capacity

E = 0.80, overall efficiency

k = 1.25, c_p/c_v , ratio of the molar heat capacities of the gas

T_{suc} = 60°F, 520°R, suction temperature

Z_{suc} = 1.0, suction compressibility factor

Pipeline Data

d = 13 inches, pipeline diameter

L = 100 miles, pipeline length

T_{avg} = 70°F = 530°R, pipeline average flowing temperature

E = 0.92, pipeline efficiency factor

P_b = 14.73 psia, base pressure

T_b = 60°F = 520°R, base temperature

P_L = 300 psia, pipeline delivery pressure

TABLE 2
GAS AND GAS-CONDENSATE PRODUCTION SYSTEM
2-1/2 INCH TUBING

<u>Constant Rate (MMscf/day)</u>	<u>Station Capacity (HP)</u>	<u>Number of Wells</u>	<u>Constant Rate (MMscf/day)</u>	<u>Station Capacity (HP)</u>	<u>Number of Wells</u>
50	3325	10	50	6650	30
100	3325	10	100	6650	30
150	3325	10	150	6650	30
200	3325	10	200	6650	30
50	3325	20	50	8300	10
100	3325	20	100	8300	10
150	3325	20	150	8300	10
200	3325	20	200	8300	10
50	3325	30	50	8300	20
100	3325	30	100	8300	20
150	3325	30	150	8300	20
200	3325	30	200	8300	20
50	6650	10	50	8300	30
100	6650	10	100	8300	30
150	6650	10	150	8300	30
200	6650	10	200	8300	30
50	6650	20	50	100	10
100	6650	20	50	500	10
150	6650	20	50	1000	10
200	6650	20	50	2000	10

TABLE 2 (Continued)
 GAS AND GAS-CONDENSATE PRODUCTION SYSTEM
 7 INCH CASING

<u>Constant Rate (MMscf/day)</u>	<u>Station Capacity (HP)</u>	<u>Number of Wells</u>	<u>Constant Rate (MMscf/day)</u>	<u>Station Capacity (HP)</u>	<u>Number of Wells</u>
50	3325	10	50	6650	30
100	3325	10	100	6650	30
150	3325	10	150	6650	30
200	3325	10	200	6650	30
50	3325	20	50	8300	10
100	3325	20	100	8300	10
150	3325	20	150	8300	10
200	3325	20	200	8300	10
50	3325	30	50	8300	20
100	3325	30	100	8300	20
150	3325	30	150	8300	20
200	3325	30	200	8300	20
50	6650	10	50	8300	30
100	6650	10	100	8300	30
150	6650	10	150	8300	30
200	6650	10	200	8300	30
50	6650	20	50	100	10
100	6650	20	50	500	10
150	6650	20	50	1000	10
200	6650	20	50	2000	10

APPENDIX B

TABLE 3
COMPARISON OF THE PERFORMANCE OF A DRY GAS PRODUCTION SYSTEM

Compressor Station Capacity:	3325 HP	6650 HP		8300 HP	
	<u>2½" Tubing</u>	<u>7" Casing</u>	<u>2½" Tubing</u>	<u>7" Casing</u>	<u>2½" Tubing</u>
Deliverability (MMCF/DAY)	118	164.7	134.82	209.8	139.45
Average Reservoir Pressure (psia)	1500	1500	1500	1500	1500
Bottom-hole Flowing Pressure (psia)	1485	1472	1481	1453	1480
Well-head Flowing Pressure (psia)	809	1263	607	1244	530
Compressor Station Suction Pressure (psia)	794	1244	581	1214	498
Compressor Station Discharge Pressure (psia)	1415	1899	1586	2367	1635
Delivery Pressure (psia)	300	300	300	300	300

TABLE 4
COMPARISON OF THE PERFORMANCE OF A DRY GAS PRODUCTION SYSTEM

	<u>Production from 10 Wells</u>	<u>Production from 20 Wells</u>	<u>Production from 30 Wells</u>
Deliverability (MMcf/day)	139.45	197.75	216.35
Average Reservoir Pressure (psia)	1500	1500	1500
Bottom-hole Flowing Pressure (psia)	1480	1490	1495
Well-head Flowing Pressure (psia)	530	977	1132
Compressor Station Suction Pressure (psia)	498	942	1096
Compressor Station Discharge Pressure (psia)	1635	2242	2433
Delivery Pressure (psia)	300	300	300

TABLE 5
COMPARISON OF THE PERFORMANCE OF A GAS-CONDENSATE PRODUCTION SYSTEM
Production Through 2-1/2" Tubing

Compressor Station Capacity:	<u>3325 HP</u>	<u>6650 HP</u>	<u>8300 HP</u>
Deliverability (MMcf/day)	23.6	33.6	35.65
Condensate Production (bbls/day)	192	274	290.9
Average Reservoir Pressure (psia)	1500	1500	1500
Bottom-hole Flowing Pressure (psia)	1499	1499	1498
Well-head Flowing Pressure (psia)	50	49.10	49.0
Compressor Station Suction Pressure (psia)	39.25	21.65	15.0
Compressor Station Discharge Pressure (psia)	431.4	524.3	544
Delivery Pressure (psia)	300	300	300

TABLE 6
 COMPARISON OF THE PERFORMANCE OF A GAS-CONDENSATE PRODUCTION SYSTEM
 $D = 2\frac{1}{2}$ " Tubing, Compressor Station Capacity = 8300 HP

	<u>Production from 10 Wells</u>	<u>Production from 20 Wells</u>	<u>Production from 30 Wells</u>
Deliverability (MMcf/day)	35.6	47.3	59
Condensate Production (bbls/day)	290.9	385.7	462
Average Reservoir Pressure (psia)	1500	1500	1500
Bottom-hole Flowing Pressure (psia)	1498	1499	1499
Well-head Flowing Pressure (psia)	49	50.2	50.5
Compressor Station Suction Pressure (psia)	15	16	16.2
Compressor Station Discharge Pressure (psia)	544	551	552
Delivery Pressure (psia)	300	300	300

TABLE 7
COMPARISON OF DRY GAS AND GAS-CONDENSATE PRODUCTION SYSTEMS
Compressor Station Capacity 3325 HP

	Production Thru 2½" Tubing		Production Thru 7" Casing	
	<u>Dry Gas</u>	<u>Gas-Condensate</u>	<u>Dry Gas</u>	<u>Gas-Condensate</u>
Deliverability (MMcf/day)	118	23.6	164.7	24
Condensate Production (bbls/day)	--	192	--	192
Average Reservoir Pressure (psia)	1500	1500	1500	1500
Bottom-hole Flowing Pressure (psia)	1485	1499	1472	1498.9
Well-head Flowing Pressure (psia)	809	50	1263	50
Compressor Station Suction Pressure (psia)	794	39.25	1244	39.25
Compressor Station Discharge Pressure (psia)	1415	431.4	1899	431.5
Delivery Pressure (psia)	300	300	300	300

TABLE 8
COMPARISON OF DRY GAS AND GAS-CONDENSATE PRODUCTION SYSTEMS
Production Thru 7" Casing
Compressor Station Capacity 8300 HP

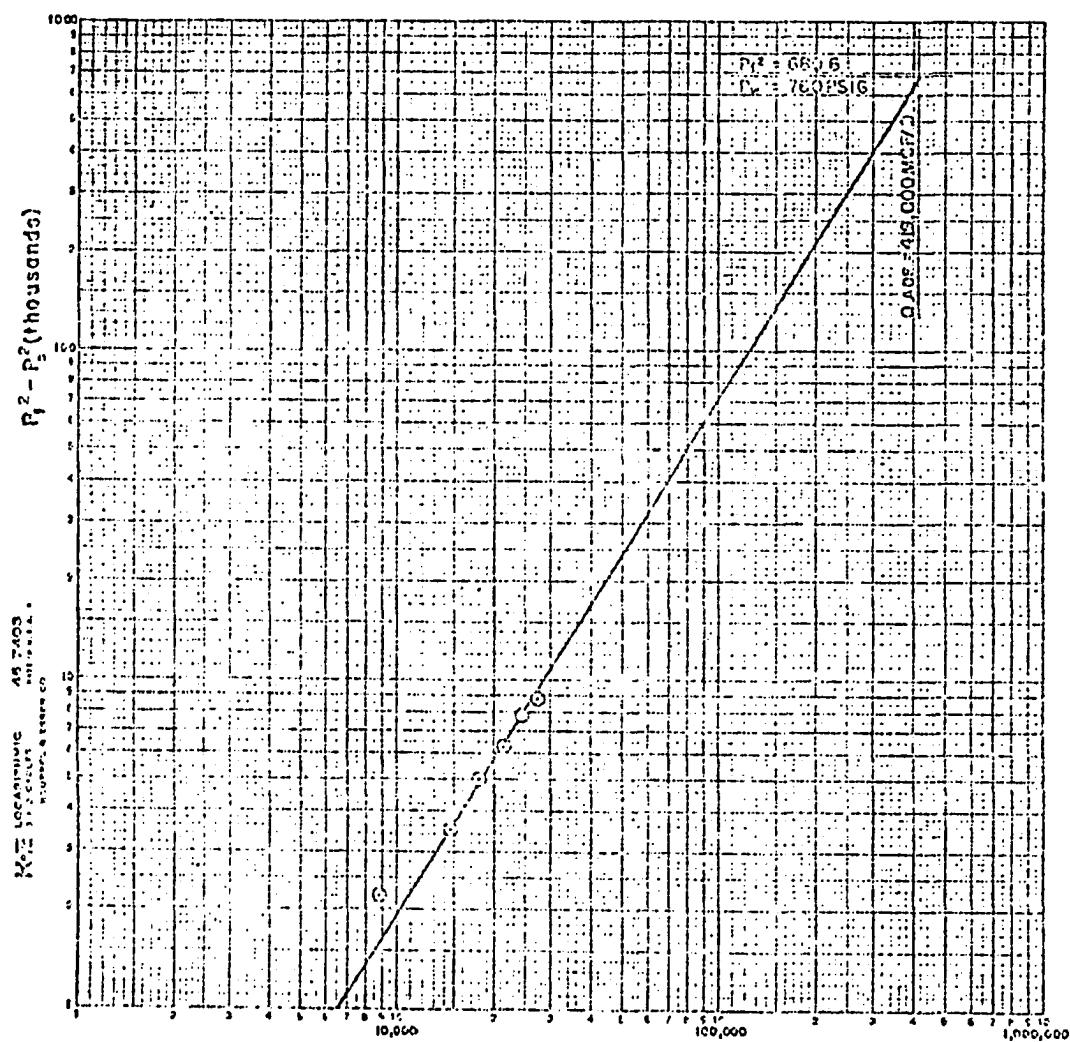
	<u>Dry Gas</u>	<u>Gas-Condensate</u>
Deliverability (MMcf/day)	228.5	35.65
Condensate Production (bbls/day)	--	290.9
Average Reservoir Pressure (psia)	1500	1500
Bottom-hole Flowing Pressure (psia)	1443	1498
Well-head Flowing Pressure (psia)	1235	49
Compressor Station Suction Pressure (psia)	1198	15
Compressor Station Discharge Pressure (psia)	2558	544
Delivery Pressure (psia)	300	300

APPENDIX C

BELLE RIVER MILLS FIELD

DOUGLASS-FRAHM #1

Tested 1/15/65

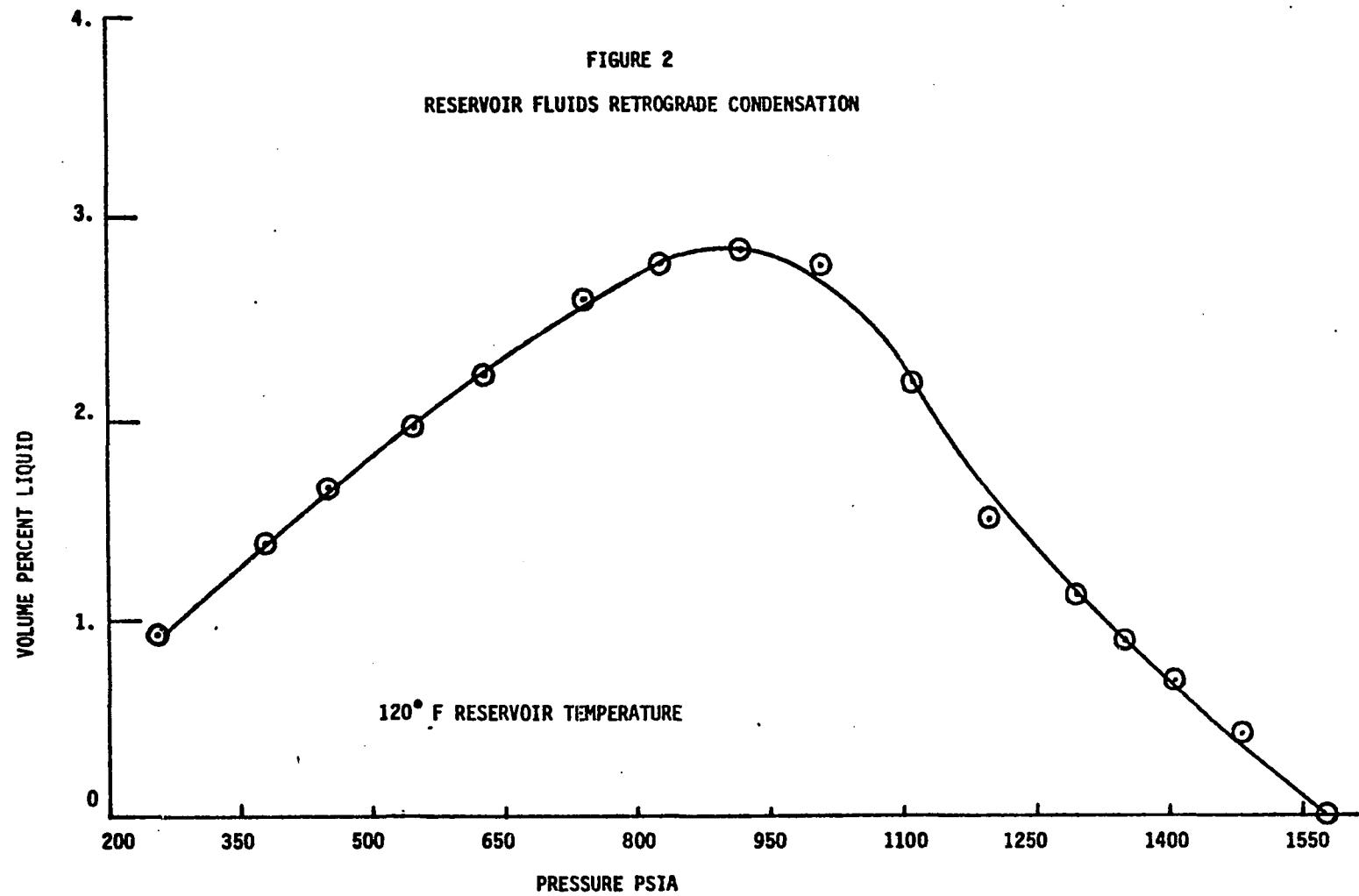


FLOWRATE - Q - MCF/D @ 15.025 PSIA

FIGURE 1

Backpressure test - Belle River Mills Field.
 (Courtesy Michigan Consolidated Gas Co.)

FIGURE 2
RESERVOIR FLUIDS RETROGRADE CONDENSATION



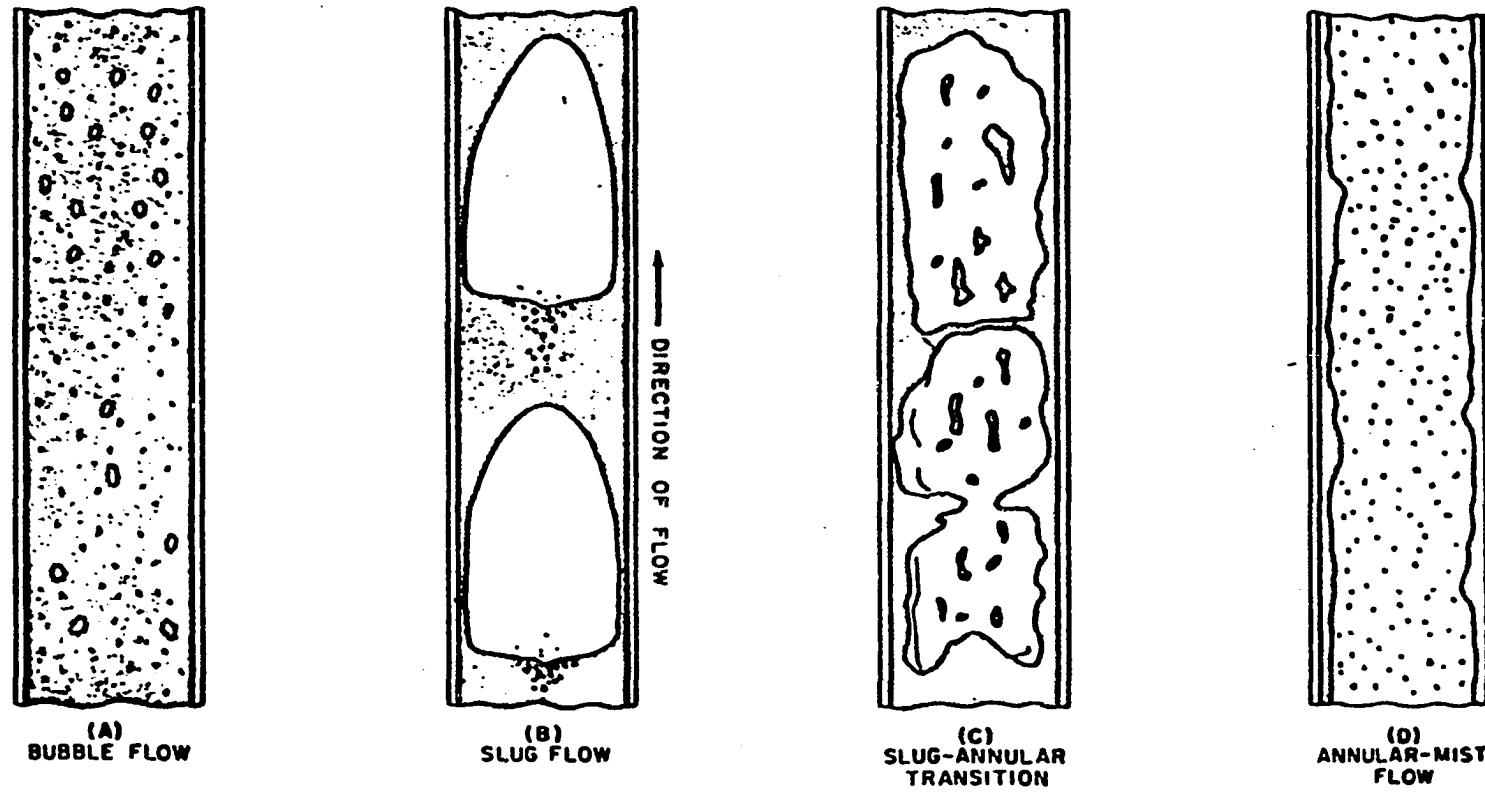


FIGURE 3. GEOMETRICAL CONFIGURATIONS IN VERTICAL FLOW.²⁹

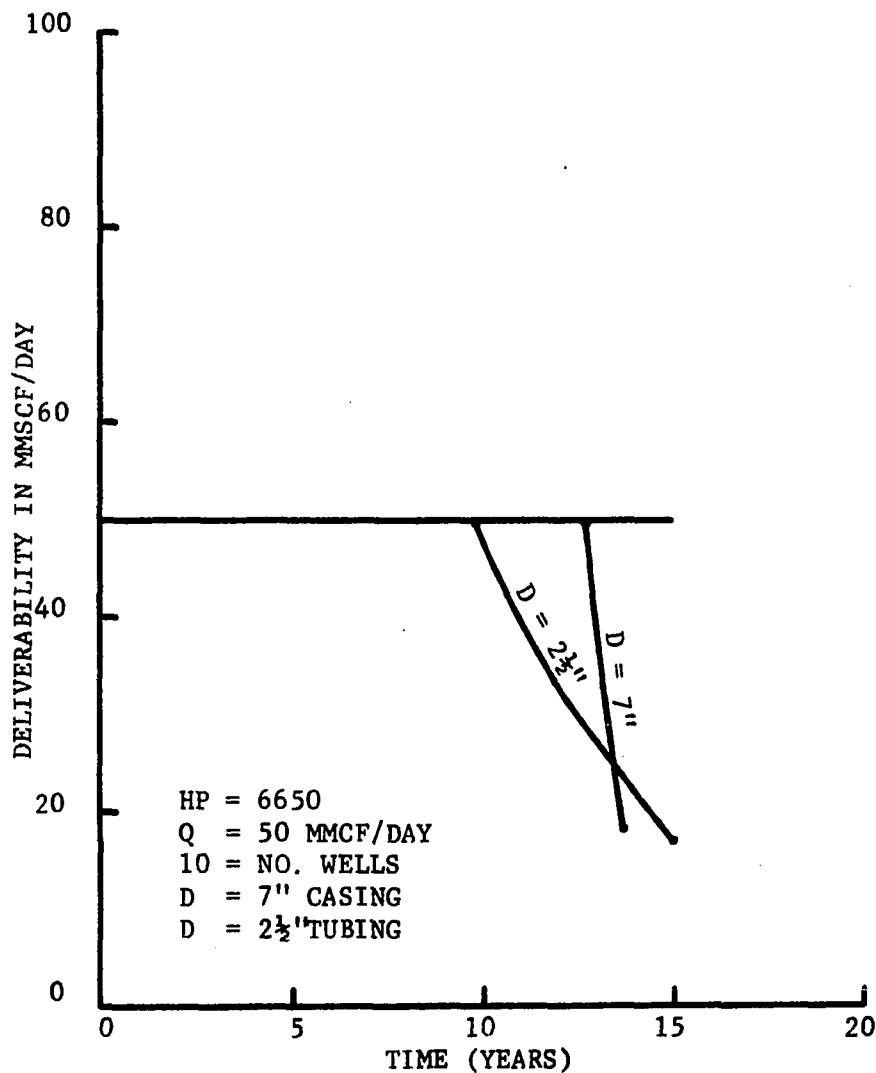


FIGURE 4. Total field deliverability vs. time
for a dry gas production system.

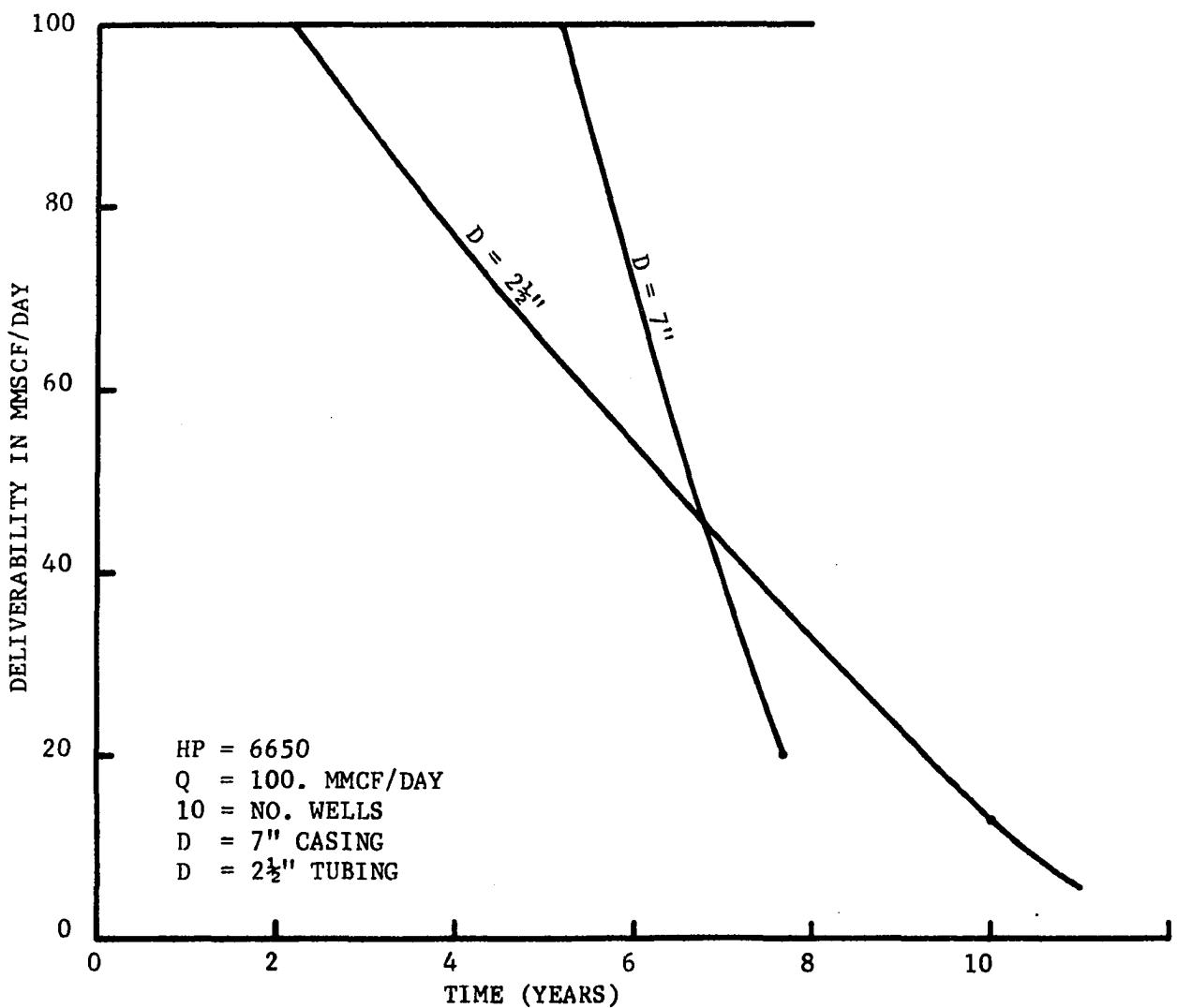


FIGURE 5. Total field deliverability vs. time
for a dry gas production system.

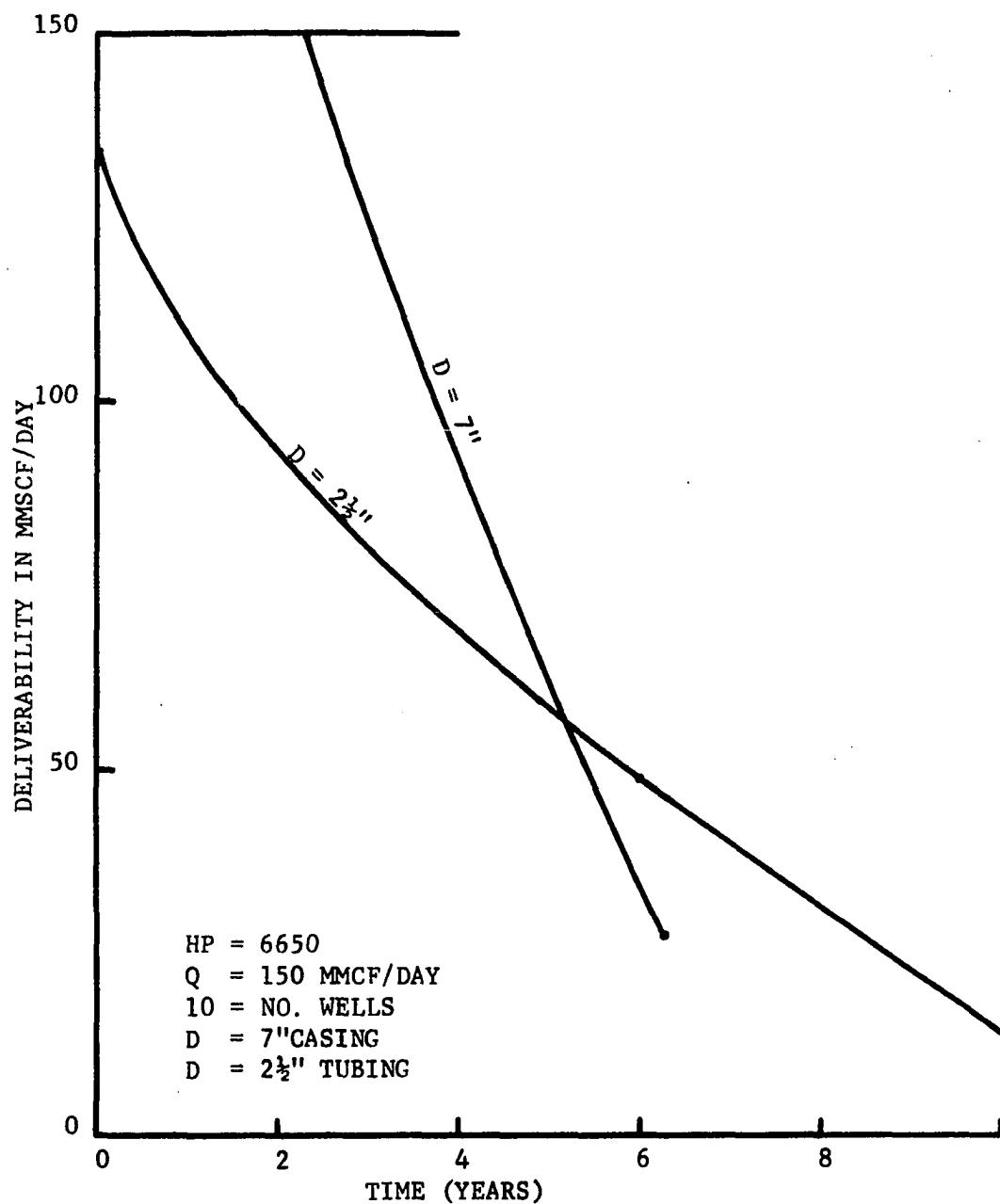


FIGURE 6. Total field deliverability vs. time
for a dry gas production system.

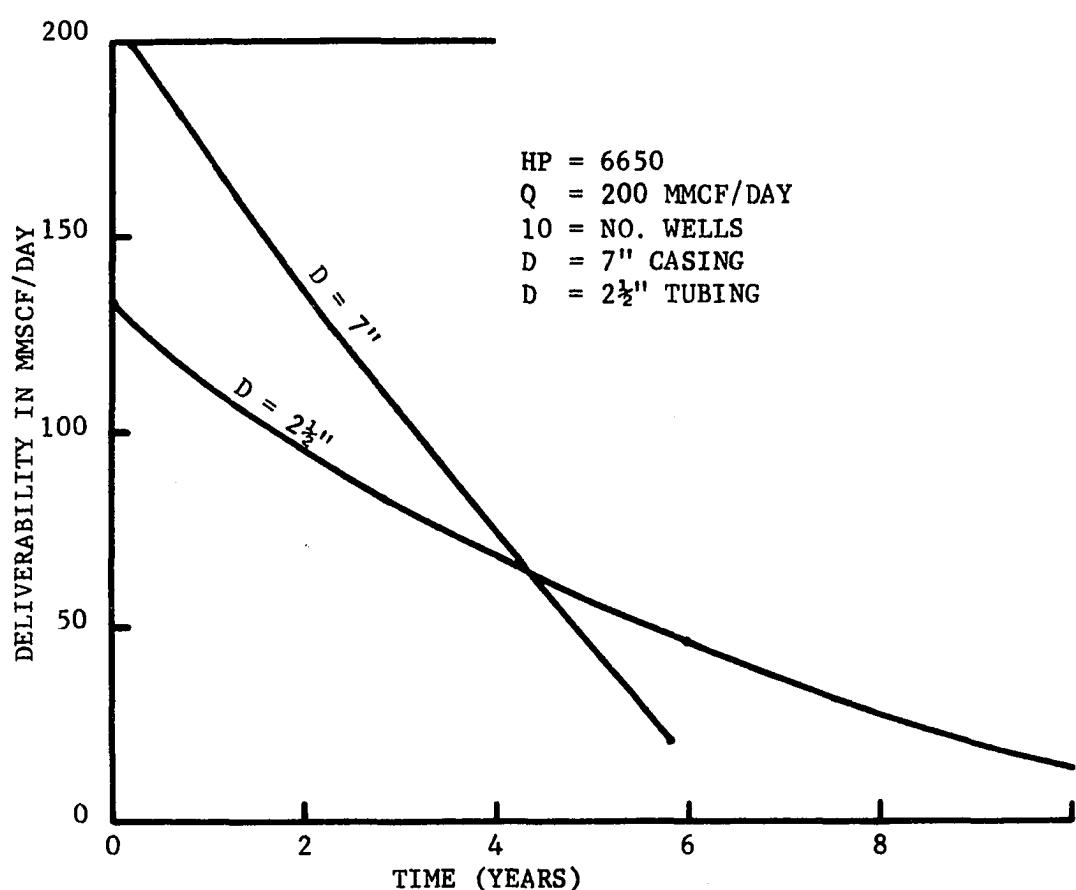


FIGURE 7. Total field deliverability vs. time
for a dry gas production system.

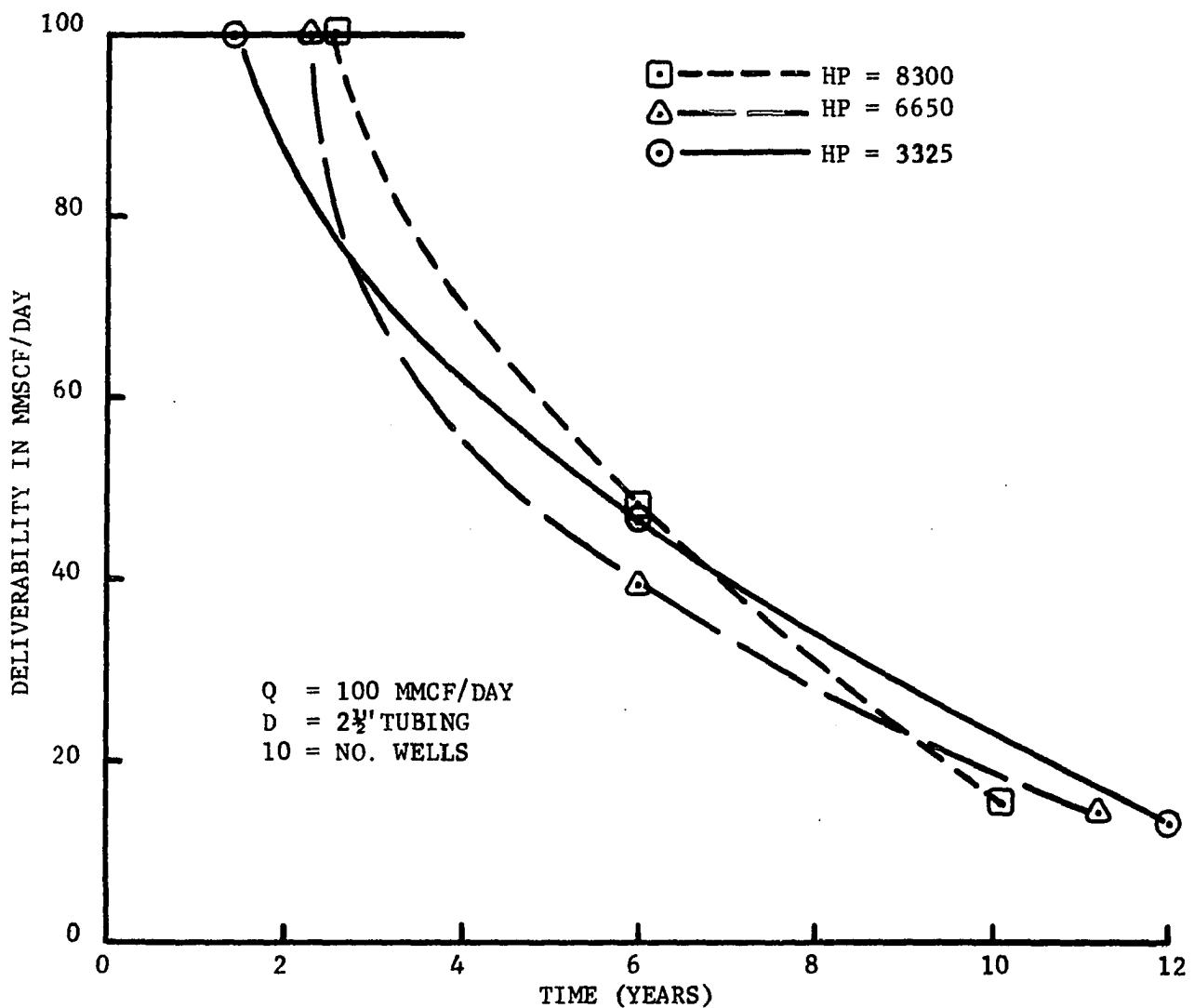


FIGURE 8. Total field deliverability and compressor station capacity for a dry gas production system.

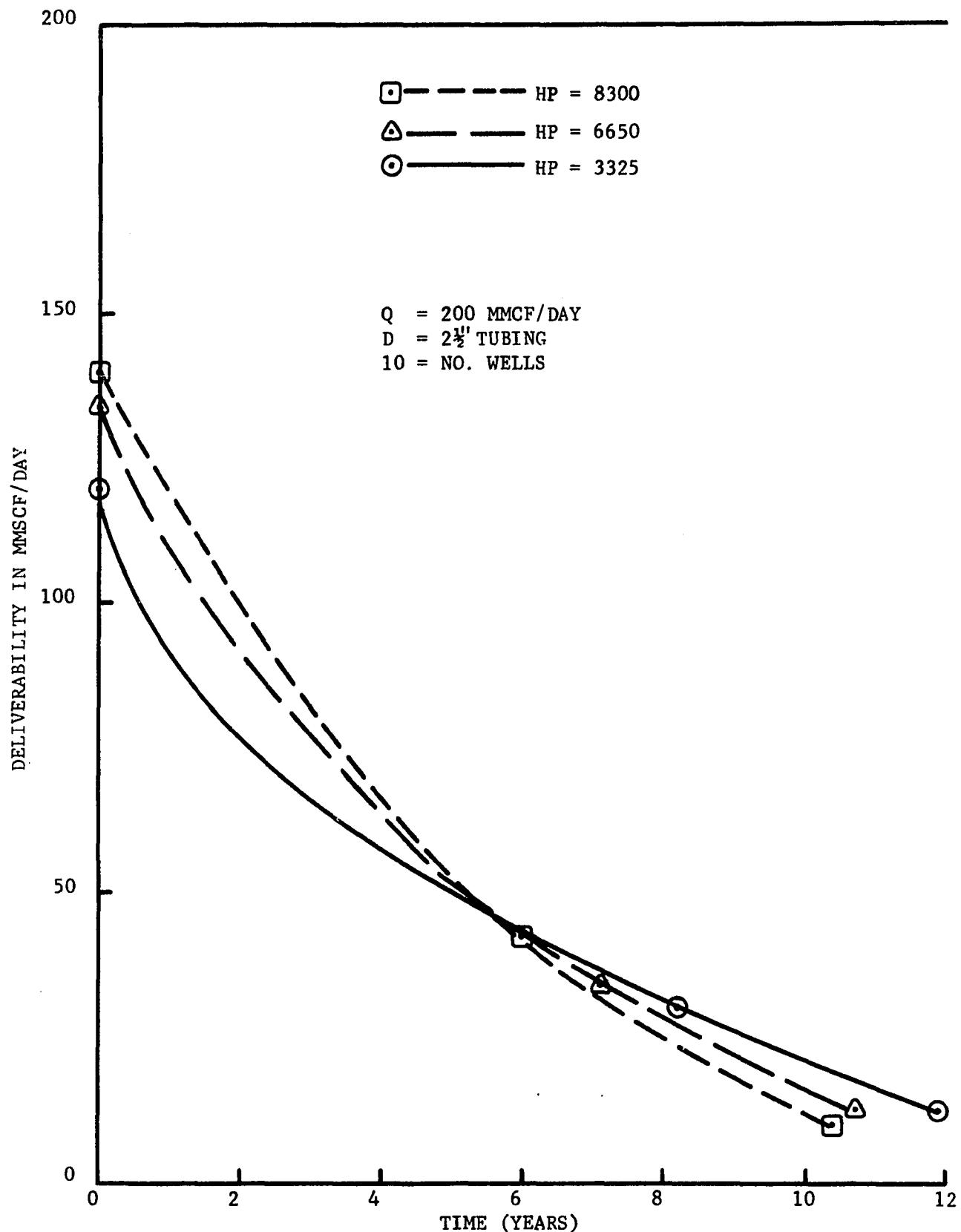


FIGURE 9. Total field deliverability and compressor station capacity for a dry gas production system.

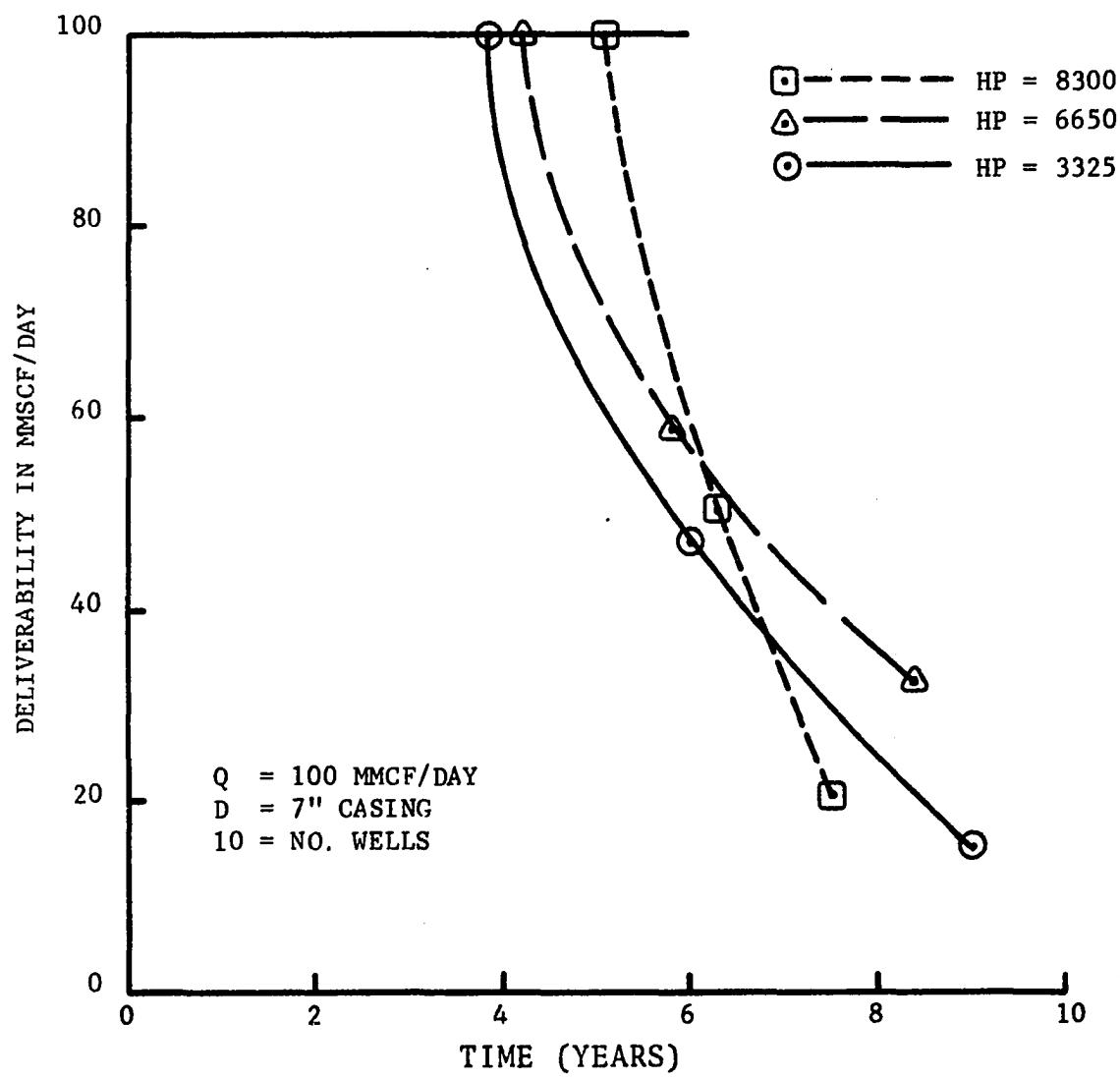


FIGURE 10. Total field deliverability and compressor station capacity for a dry gas production system.

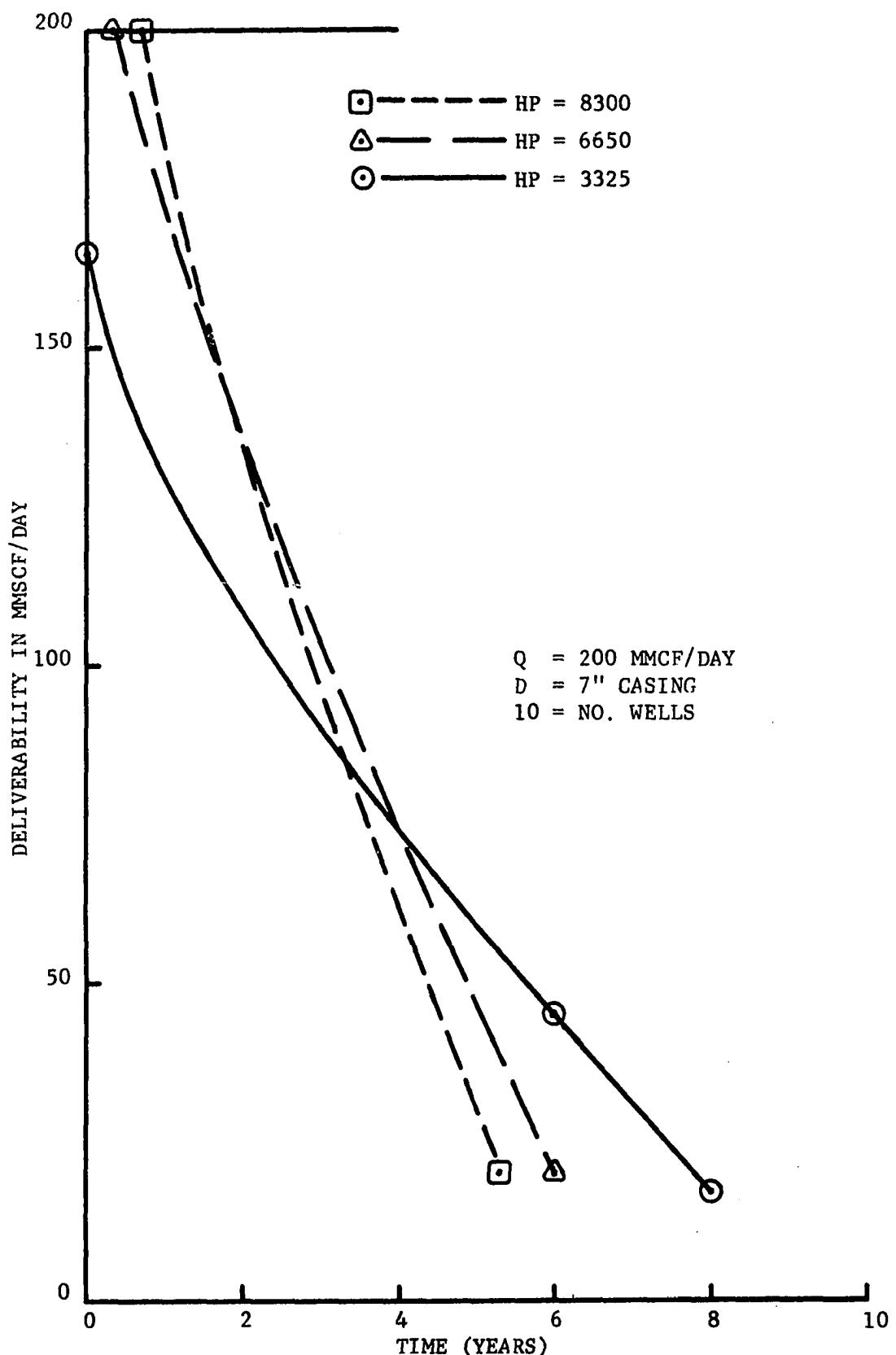


FIGURE 11. Total field deliverability and compressor station capacity for a dry gas production system.

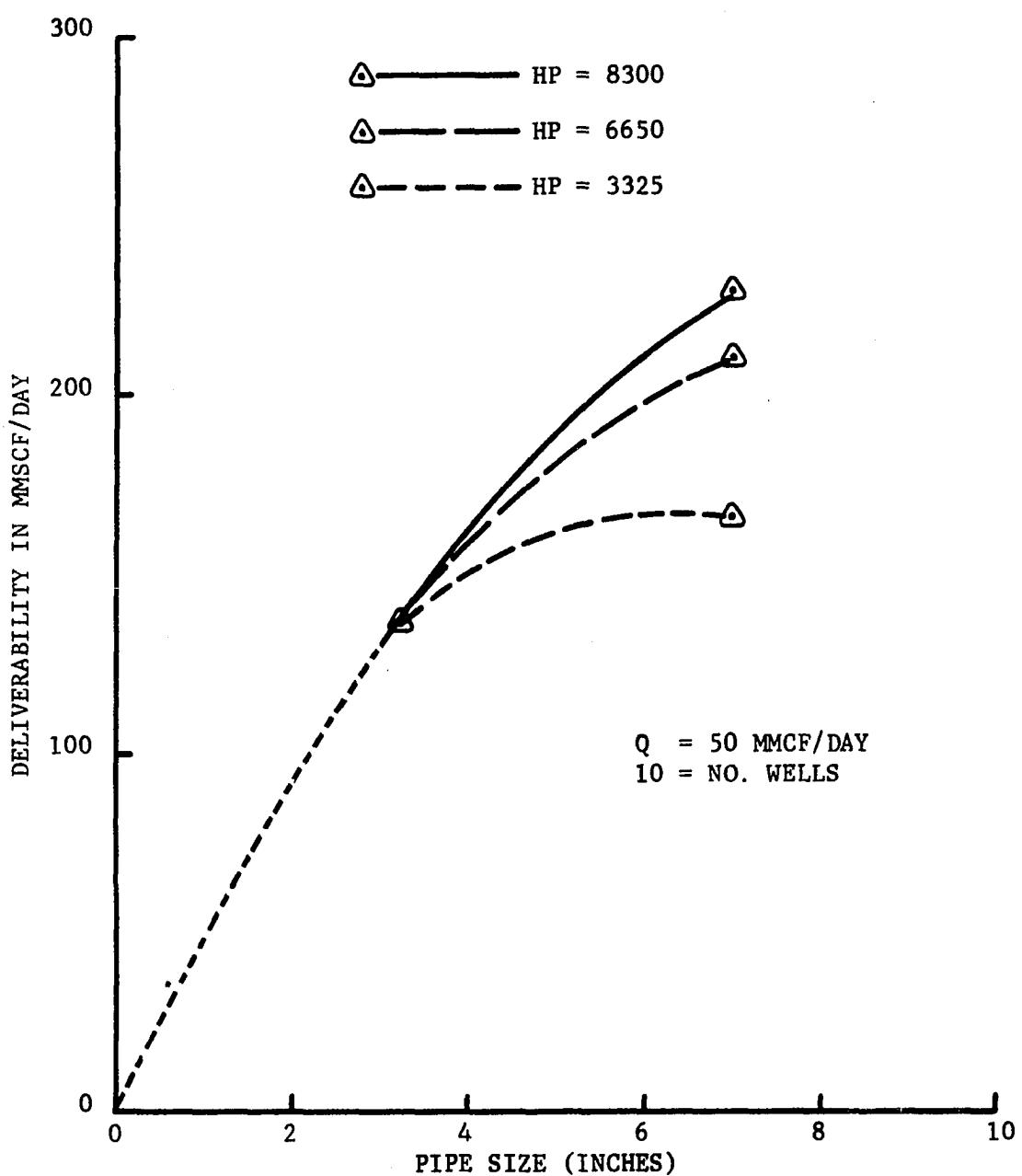


FIGURE 12. Total field deliverability vs. the size of the producing tubing strings for a dry gas production system.

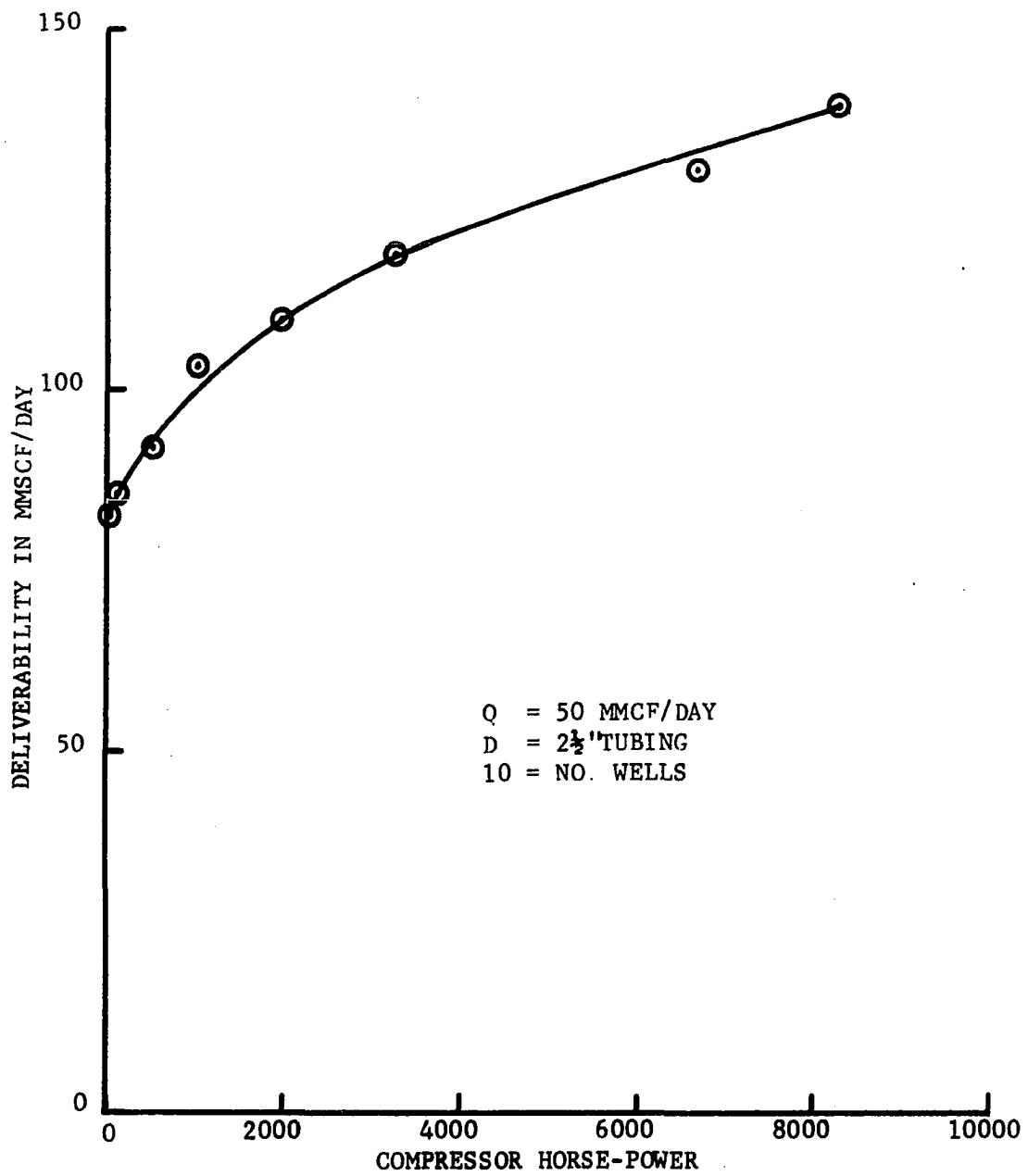


FIGURE 13. Total field deliverability vs. compressor station capacity for a dry gas production system.

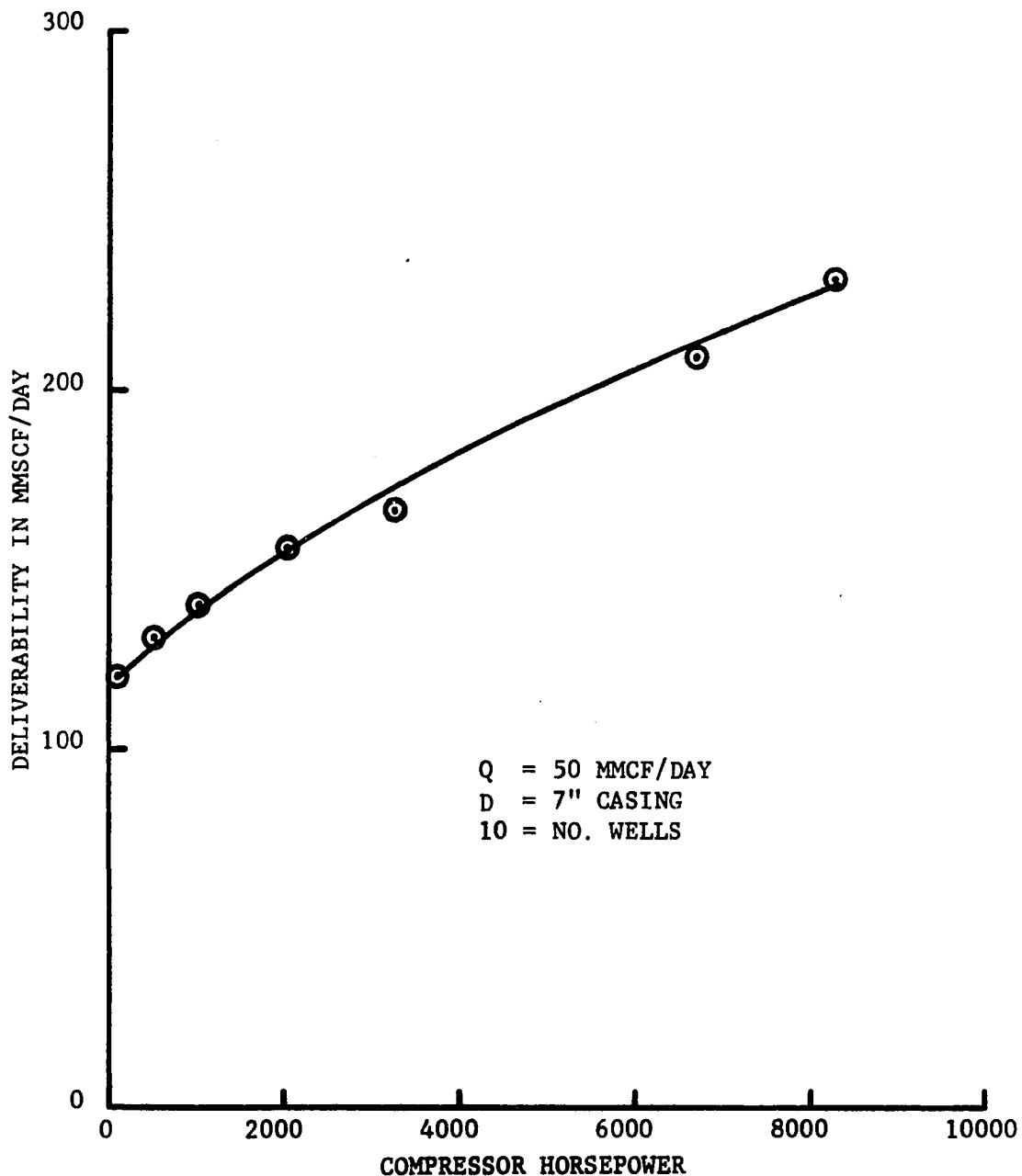


FIGURE 14. Total field deliverability vs. compressor station capacity for a dry gas production system.

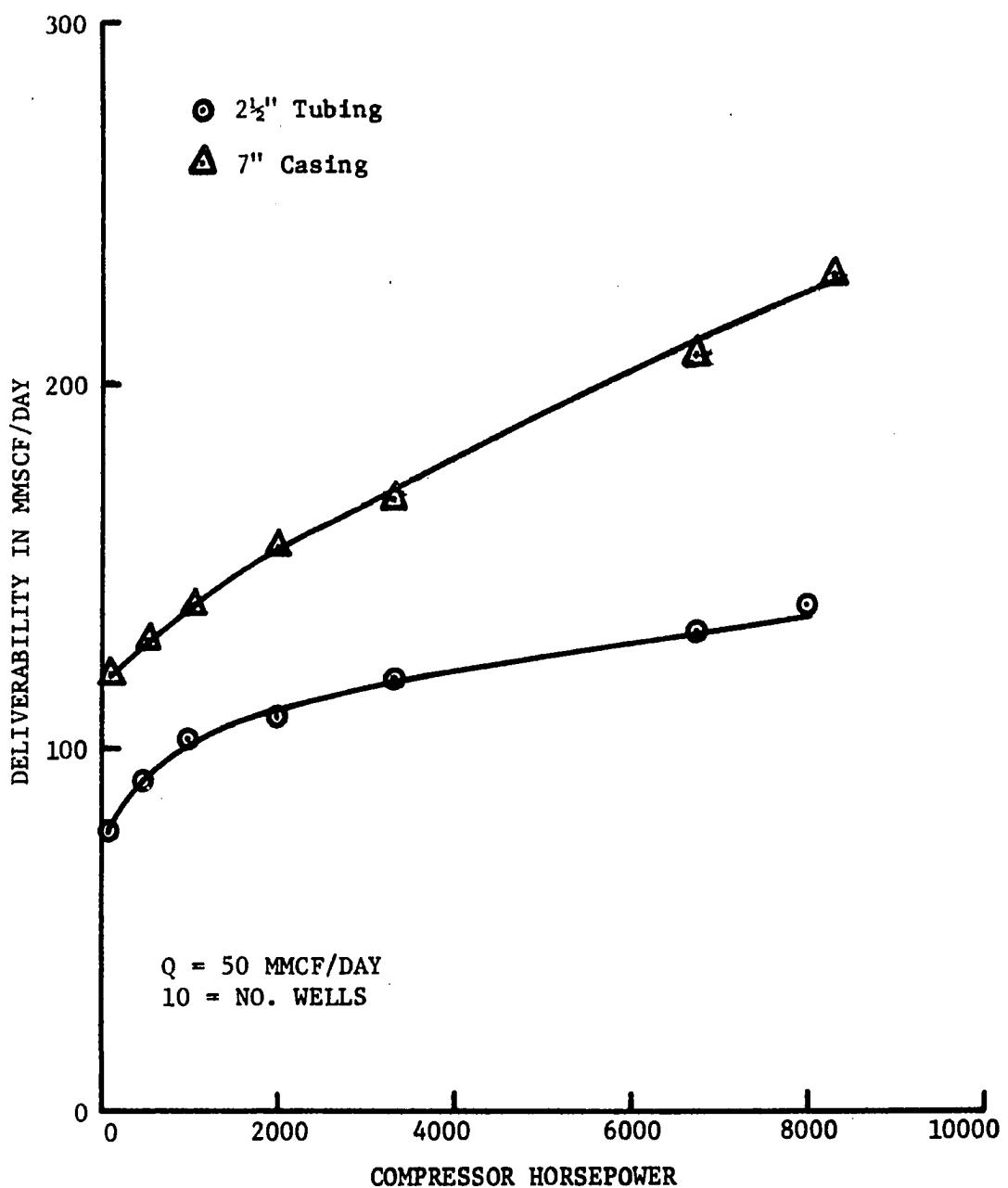


FIGURE 15. Comparison of the total field deliverability vs. the compressor station capacity for a dry gas production system.

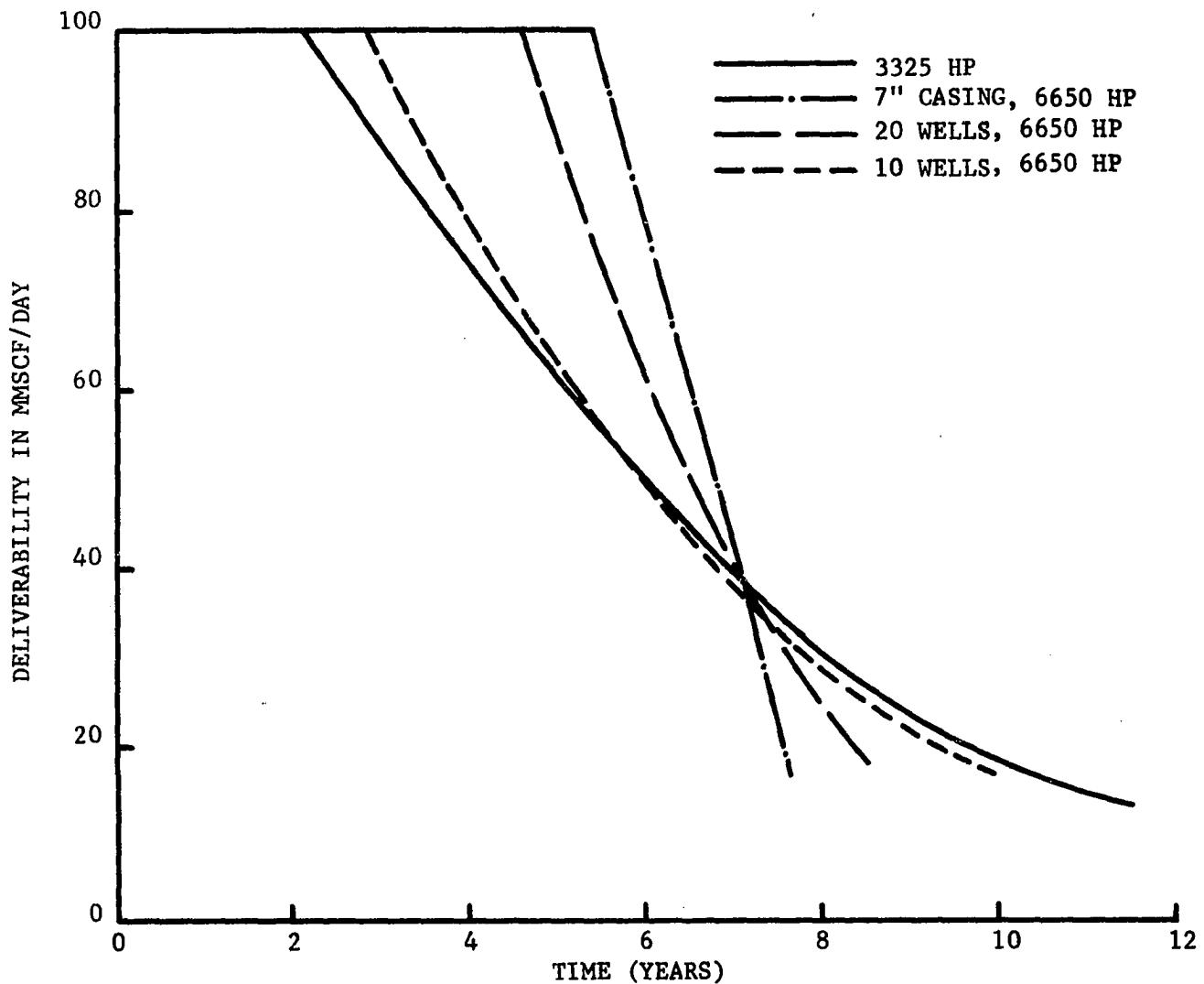


FIGURE 16. Comparison of the total field deliverability
vs. time for a dry gas production system.

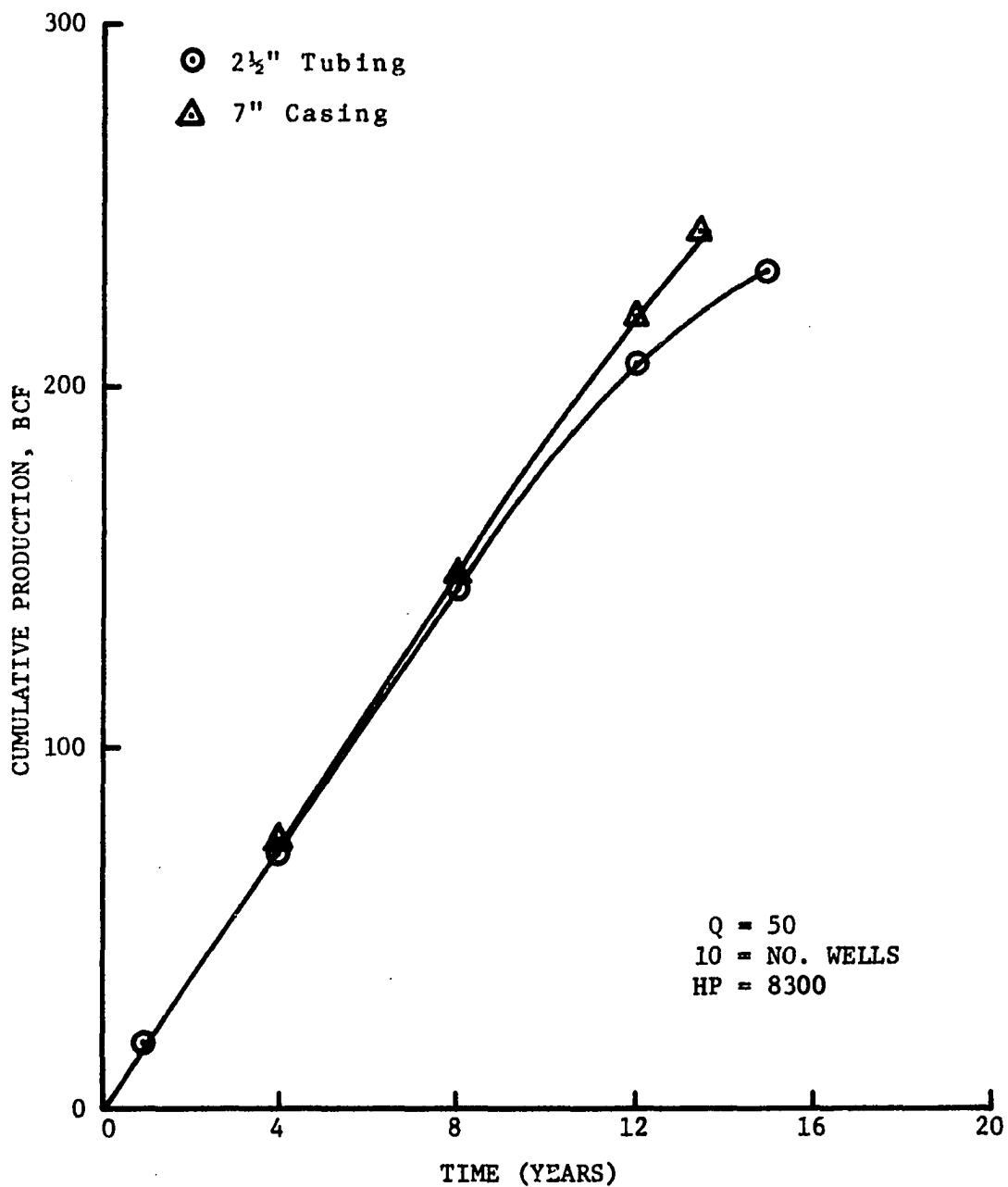
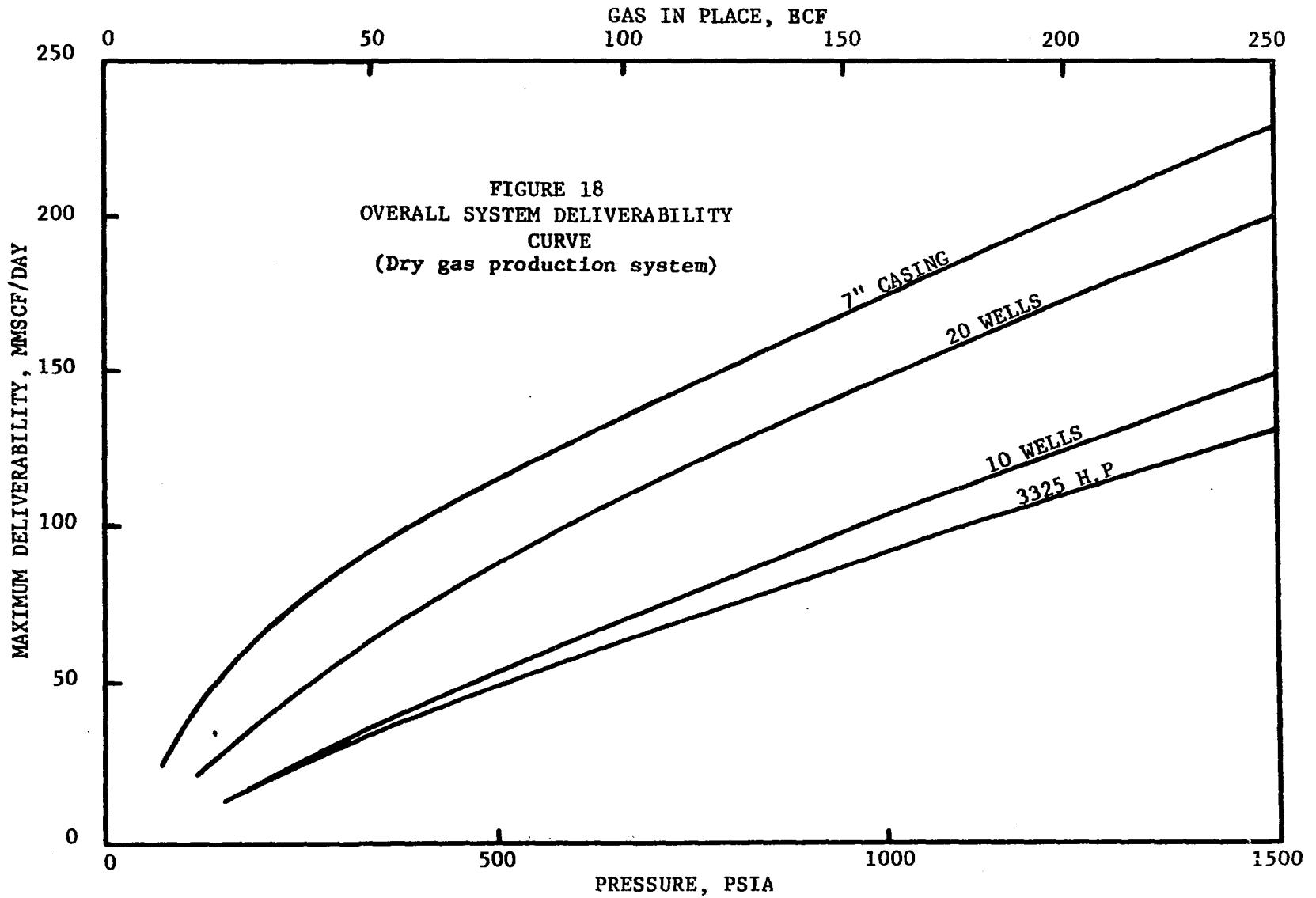


FIGURE 17. Comparison of the cumulative production vs. time for a dry gas production system.



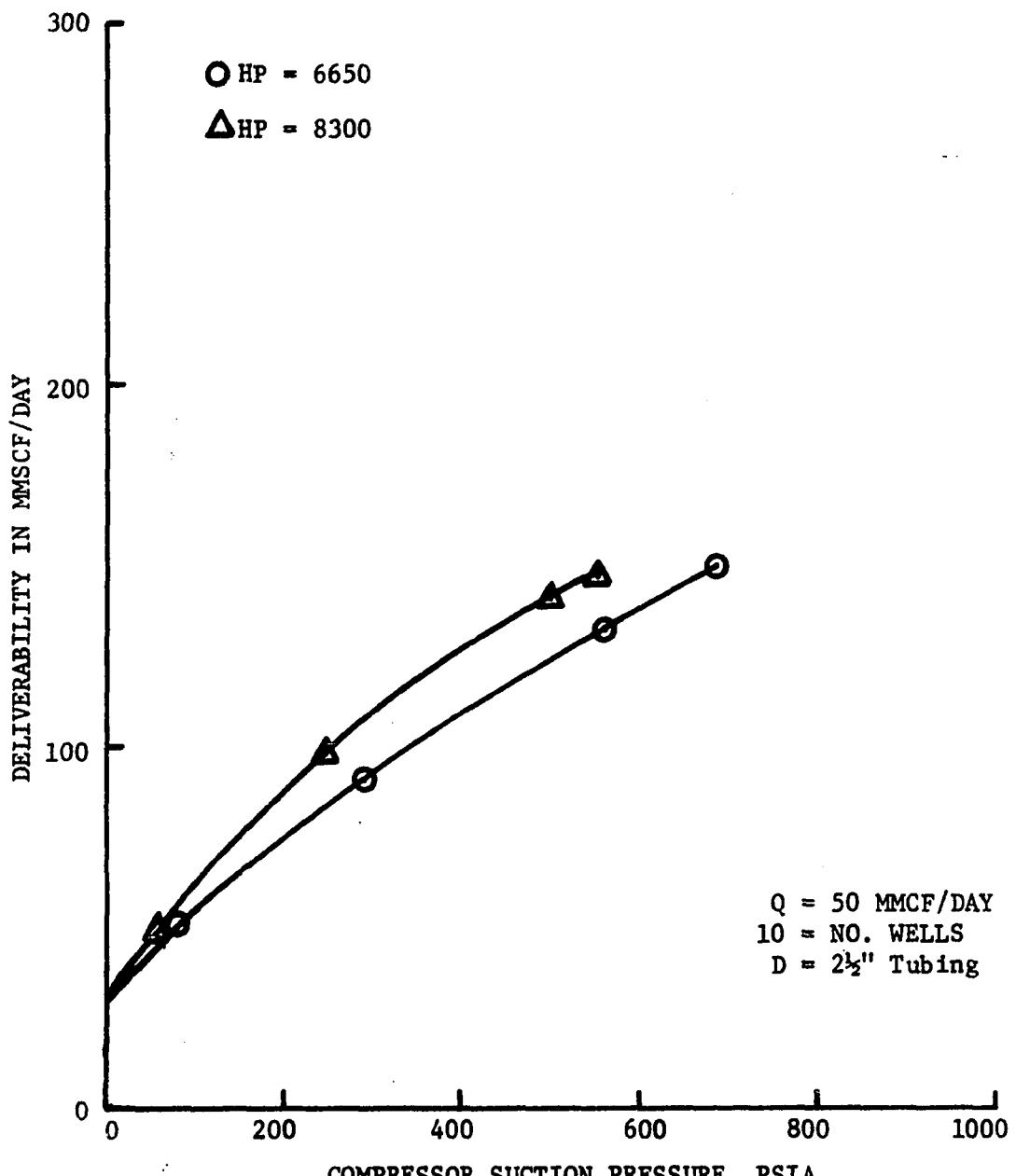


FIGURE 19. Total field deliverability vs. compressor station suction pressure for a dry gas production system.

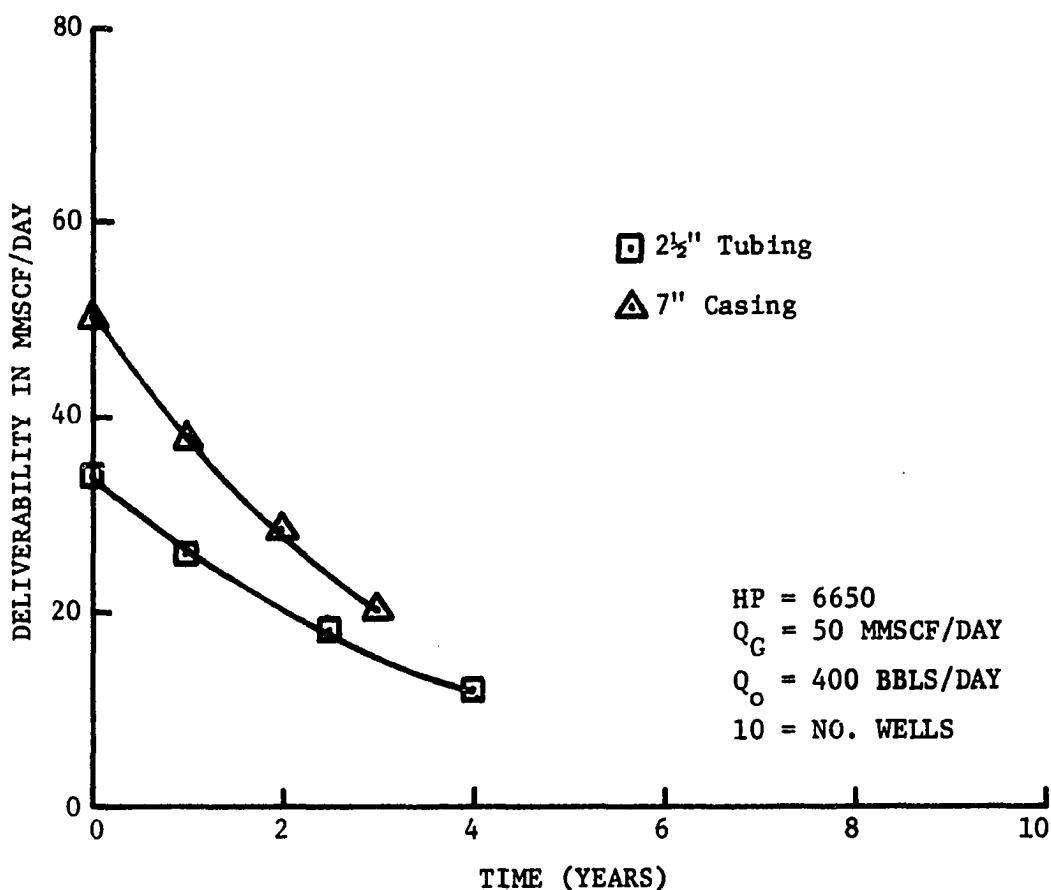


FIGURE 20. Total field deliverability vs. time

for a gas-condensate production system.

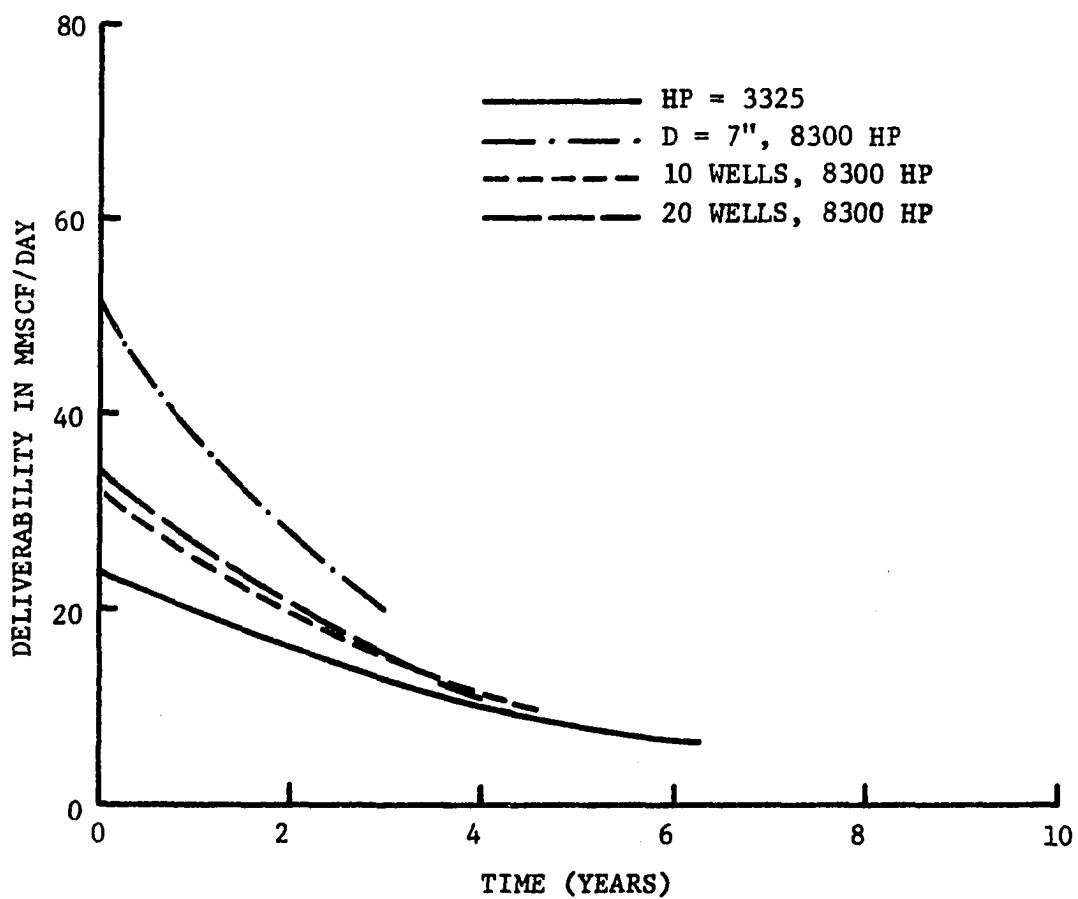


FIGURE 21. Total field deliverability vs. time
for a gas-condensate production system.

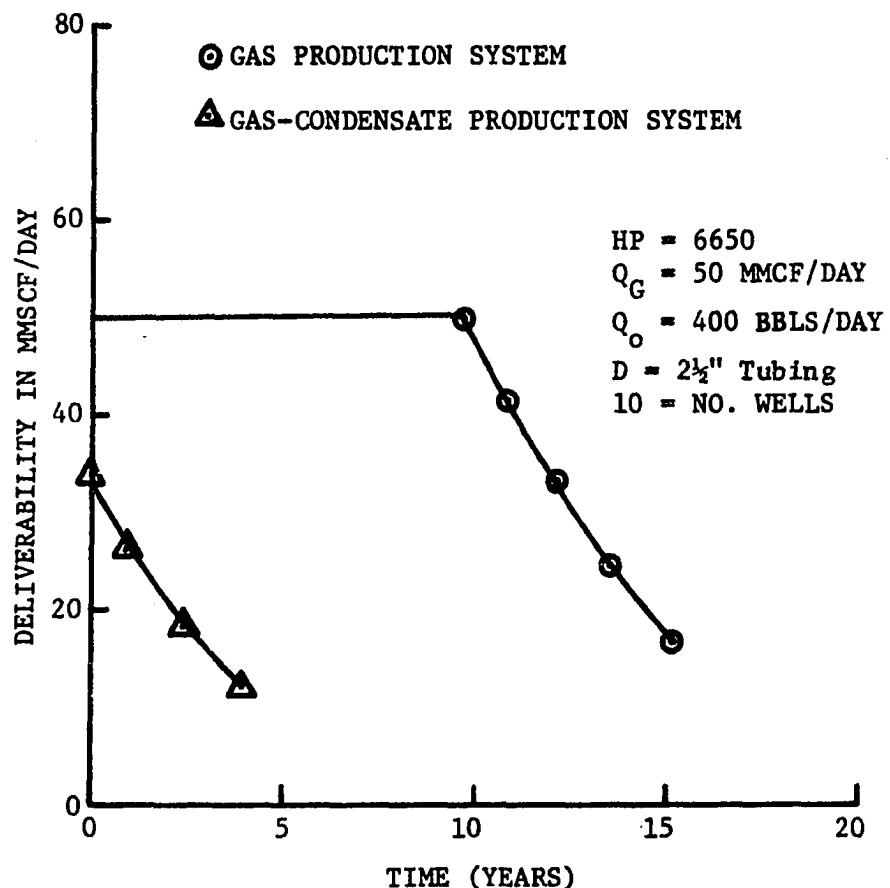


FIGURE 22. Comparison of the total field deliverability of a dry gas and a gas-condensate system vs. time.

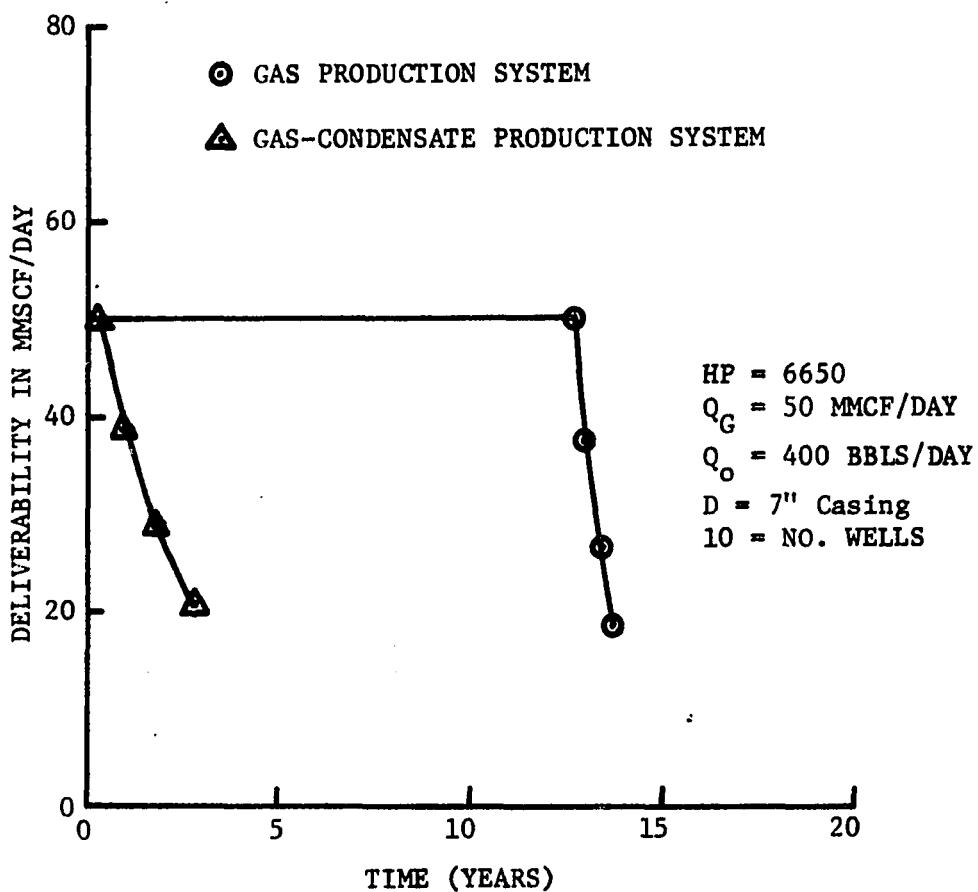


FIGURE 23. Comparison of the total field deliverability of a dry gas and a gas-condensate system vs. time.

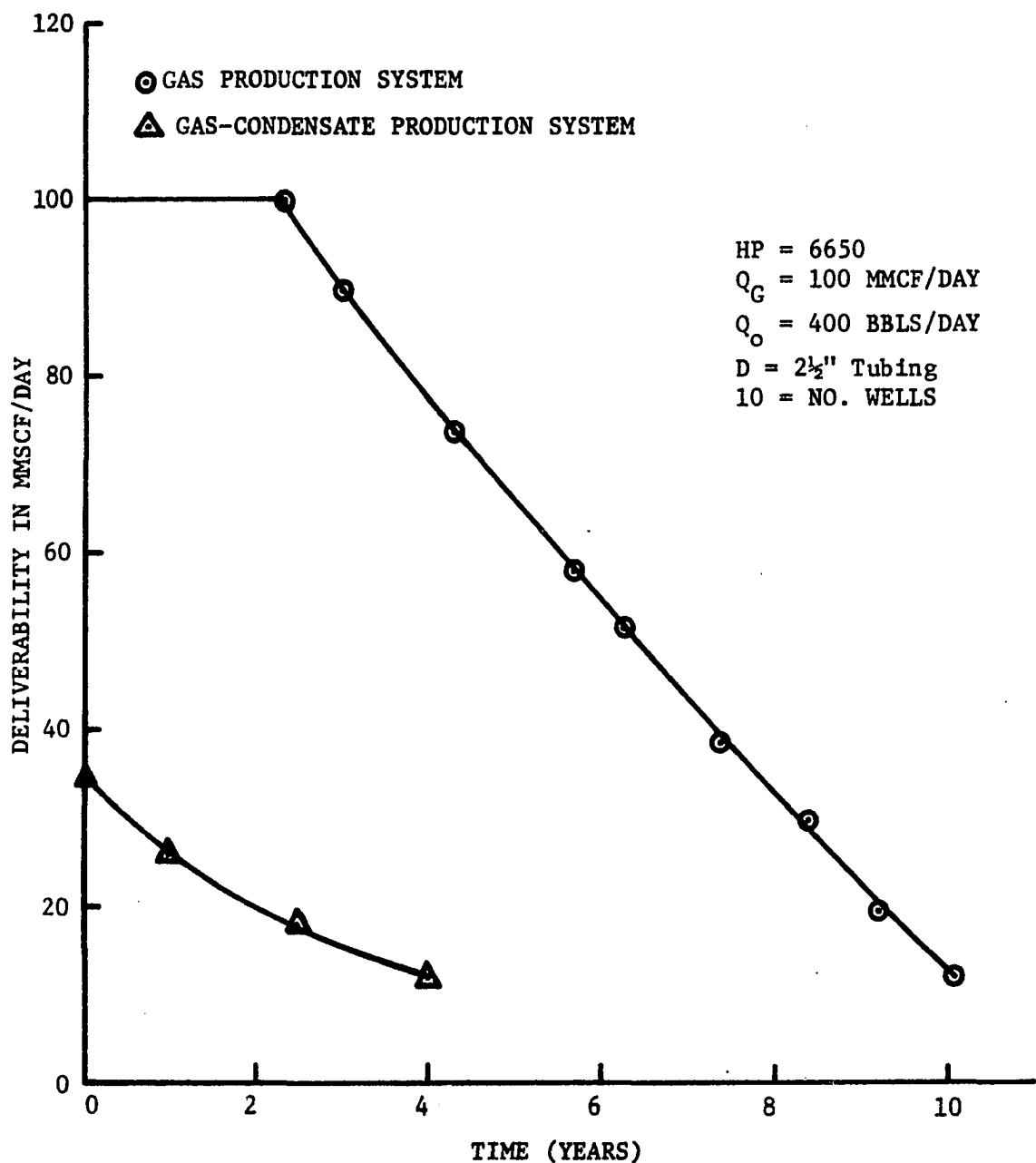


FIGURE 24. Comparison of the total field deliverability of a dry gas and a gas-condensate system vs. time.

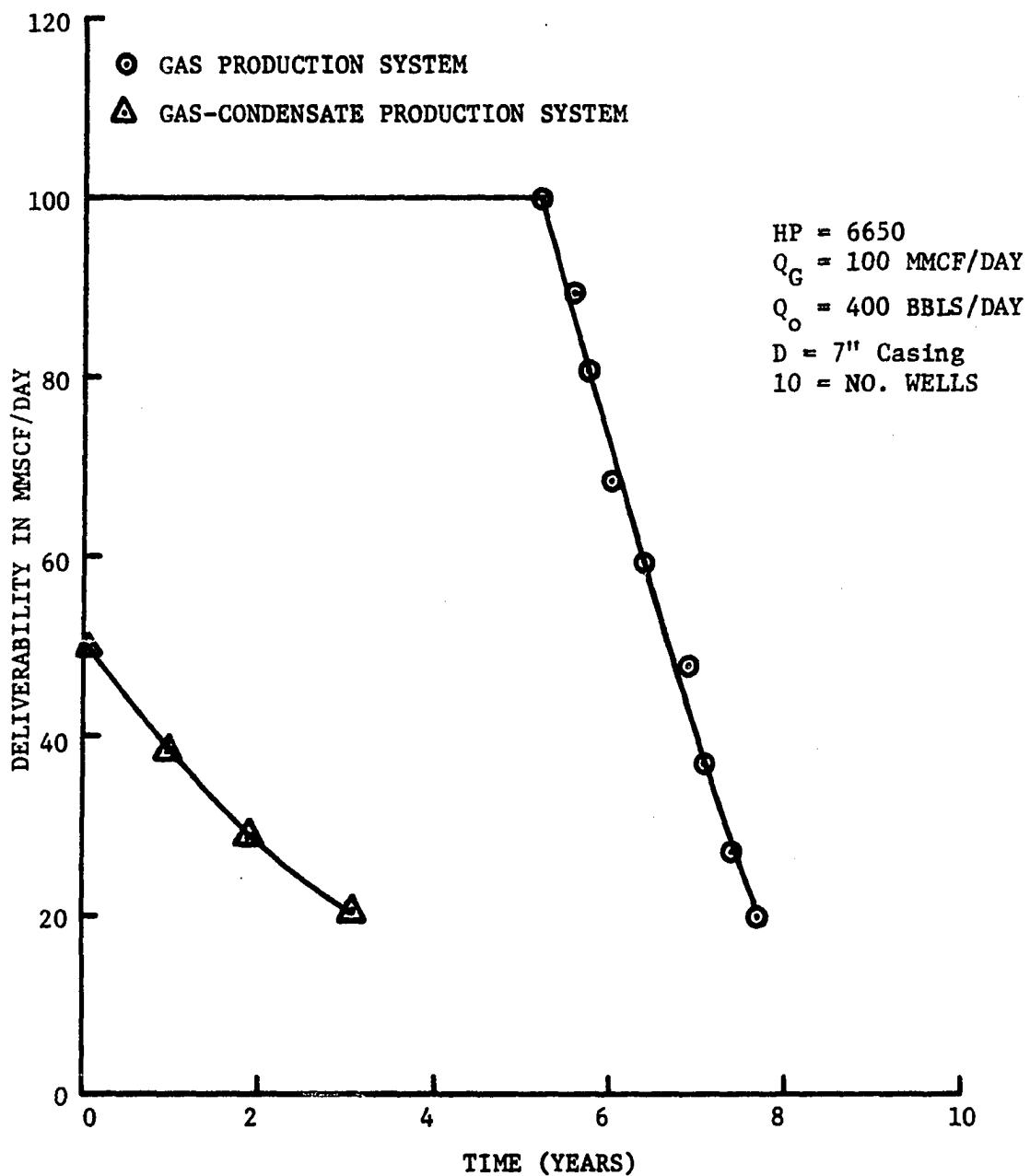


FIGURE 25. Comparison of the total field deliverability
of a dry gas and a gas-condensate system vs. time.

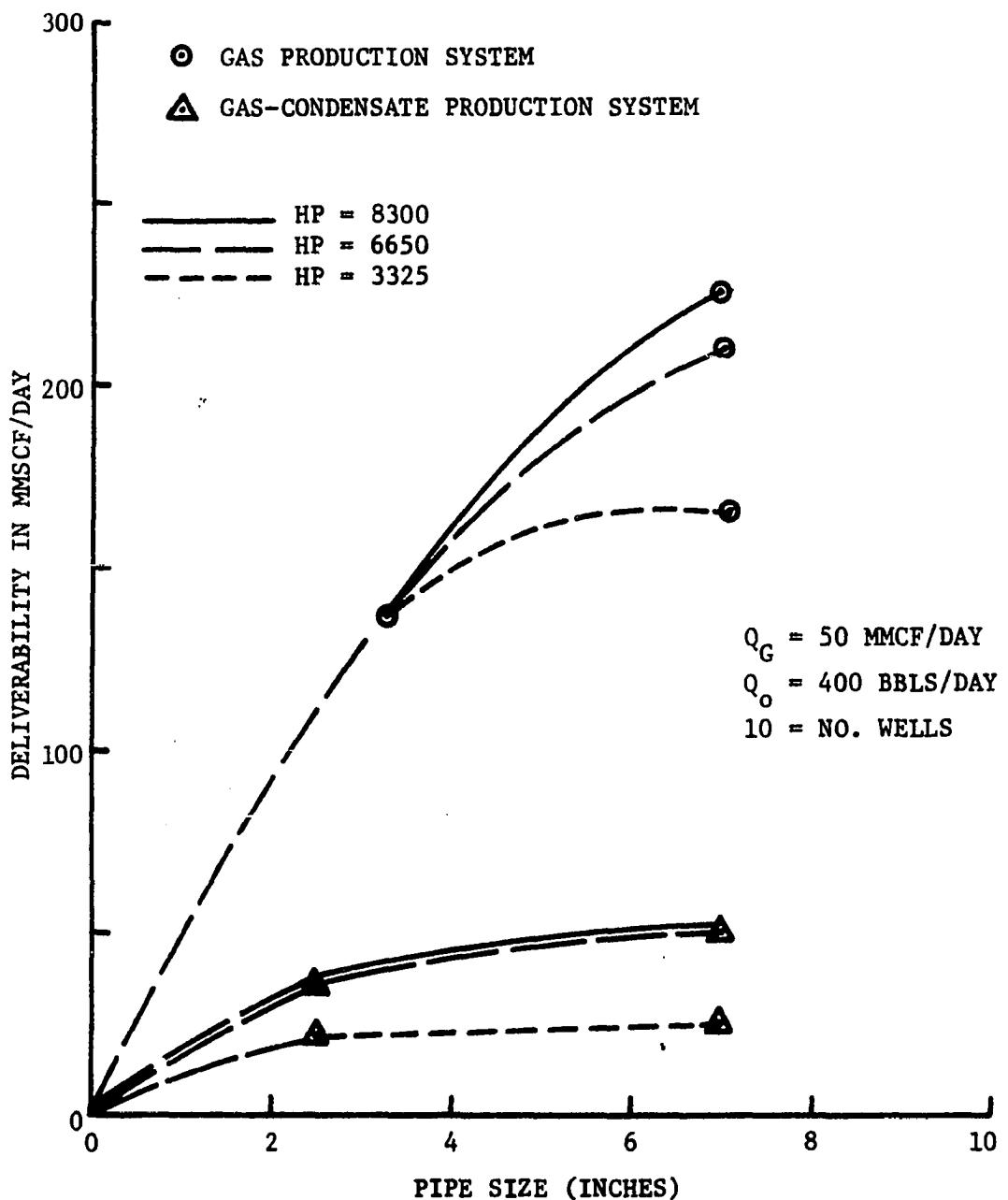


FIGURE 26. Comparison of total field deliverability of a dry gas and gas condensate system vs. the size of tubing strings.

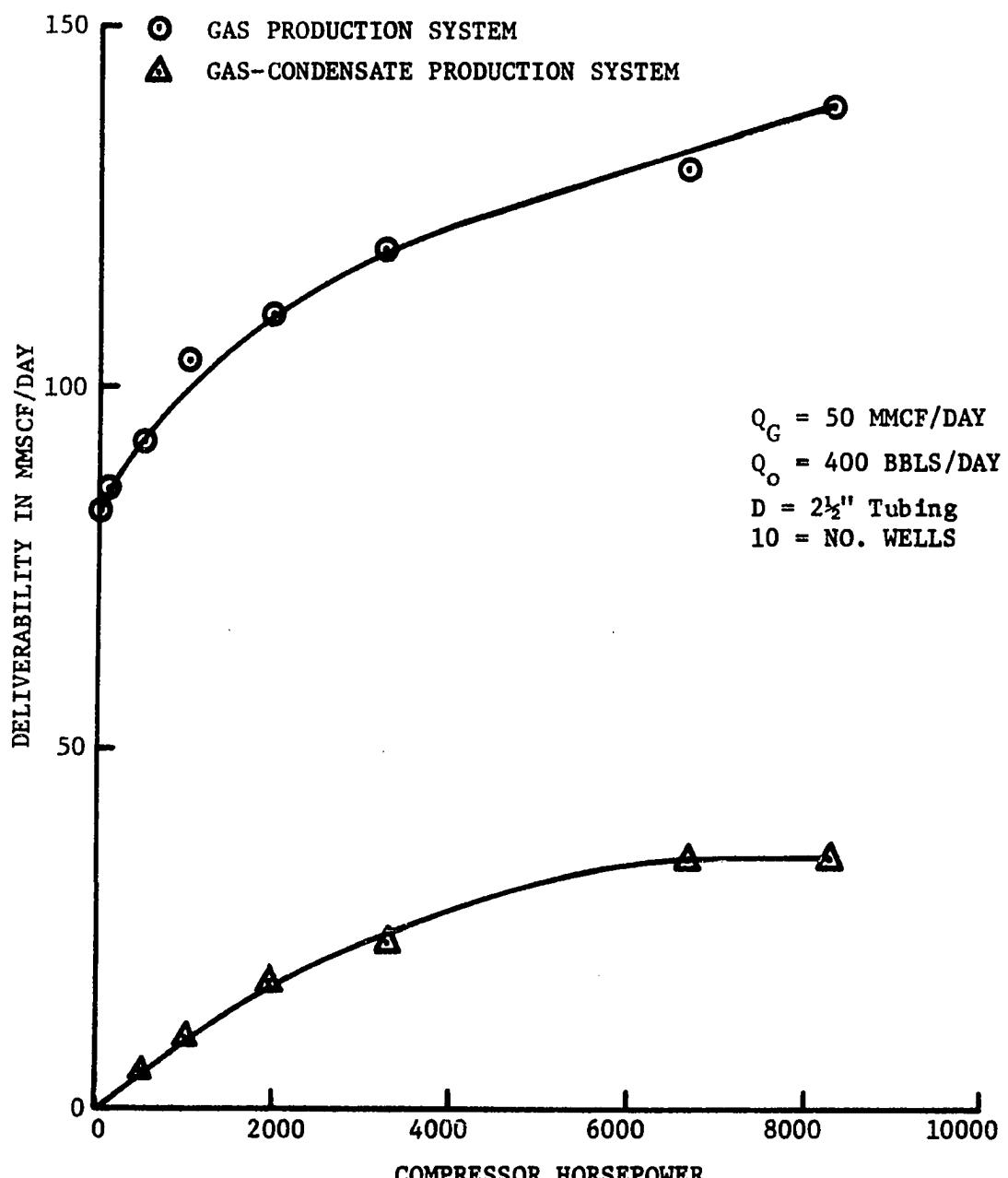


FIGURE 27. Comparison of the total field deliverability of a gas and a gas-condensate system vs. the compressor station capacity.

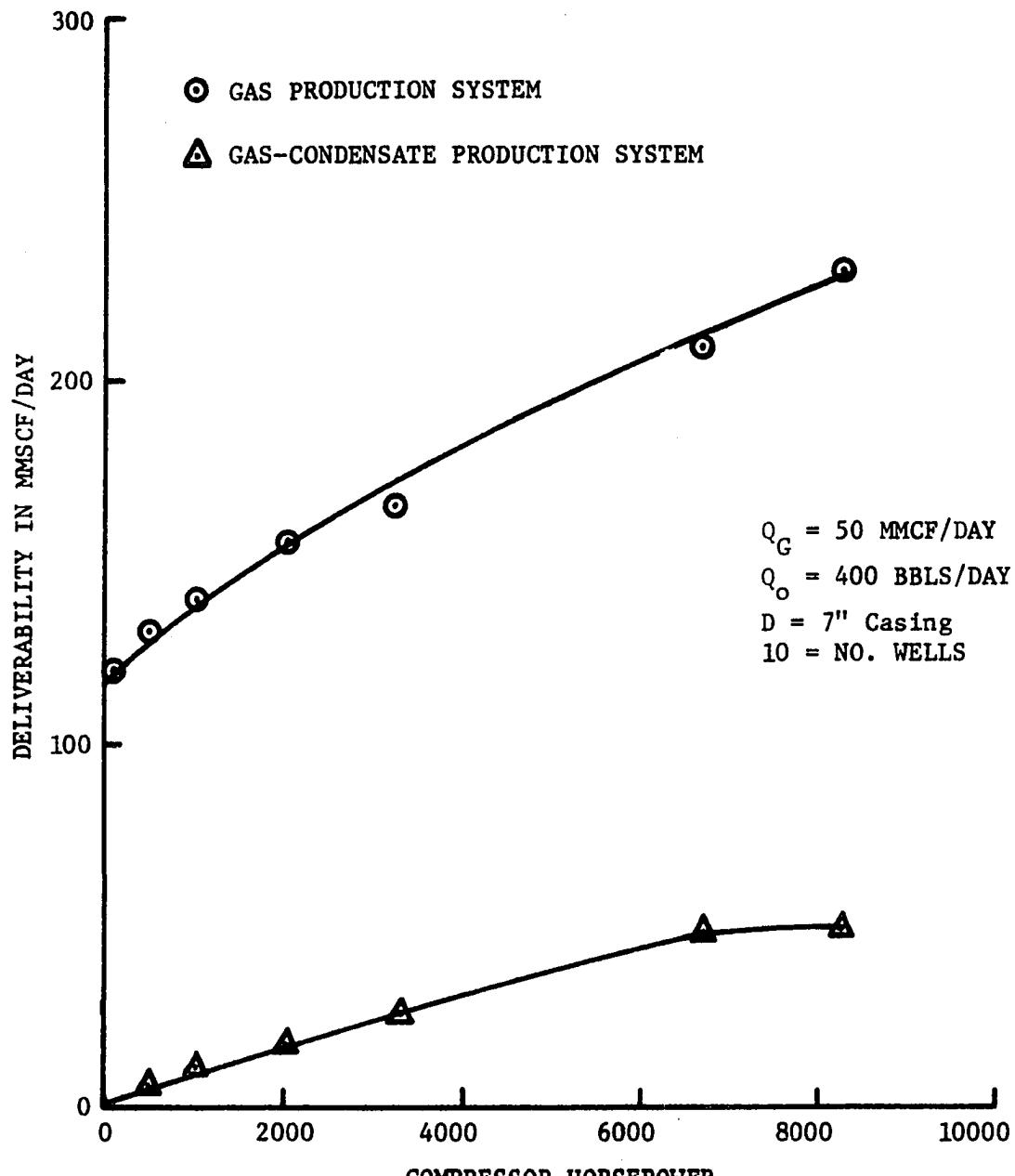


FIGURE 28. Comparison of the total field deliverability of a gas and a gas-condensate system vs. the compressor station capacity.

APPENDIX D

CALCULATION OF CONDENSATE RATE

The prediction of the condensate rate was based on Figure 2. Figure 2, which shows the percent liquid at the reservoir condition, is the result of laboratory analysis on the bottom-hole or wellhead flowing sample. If we assume that the GOR at any value of gas rate and reservoir pressure is some ratio of the initial GOR, we can calculate the condensate rate for any time step as follows:

$$(GOR)_J = GOR' \times \frac{C'}{C}$$

where:

GOR' = initial gas-condensate ratio

C' = the % liquid at initial reservoir pressure

C = the % liquid at (reservoir pressure)_J

Therefore,

$$\frac{(Q_g)_J}{(Q_o)_J} = \frac{Q'_g}{Q'_o} \times \frac{C'}{C}$$

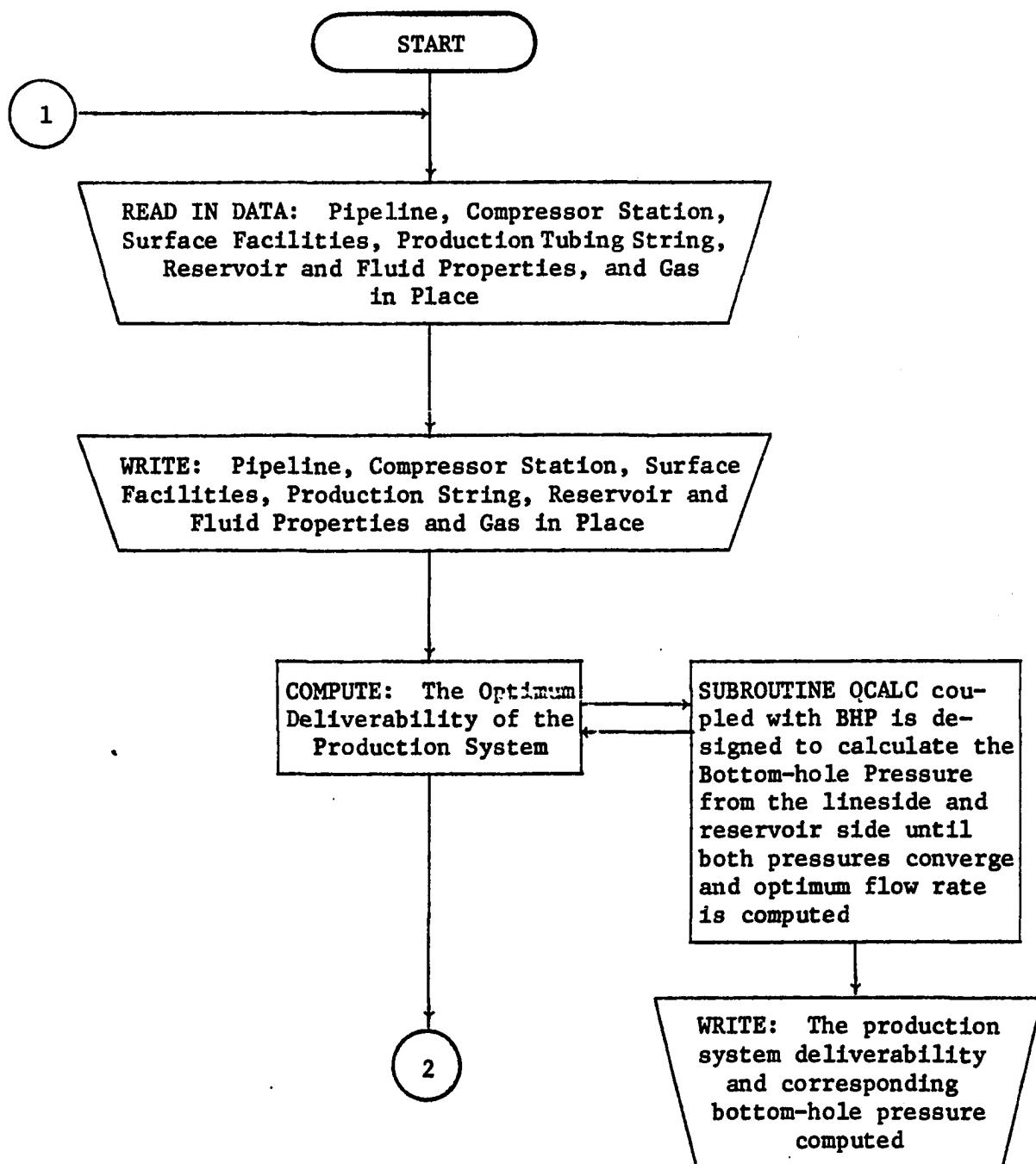
or

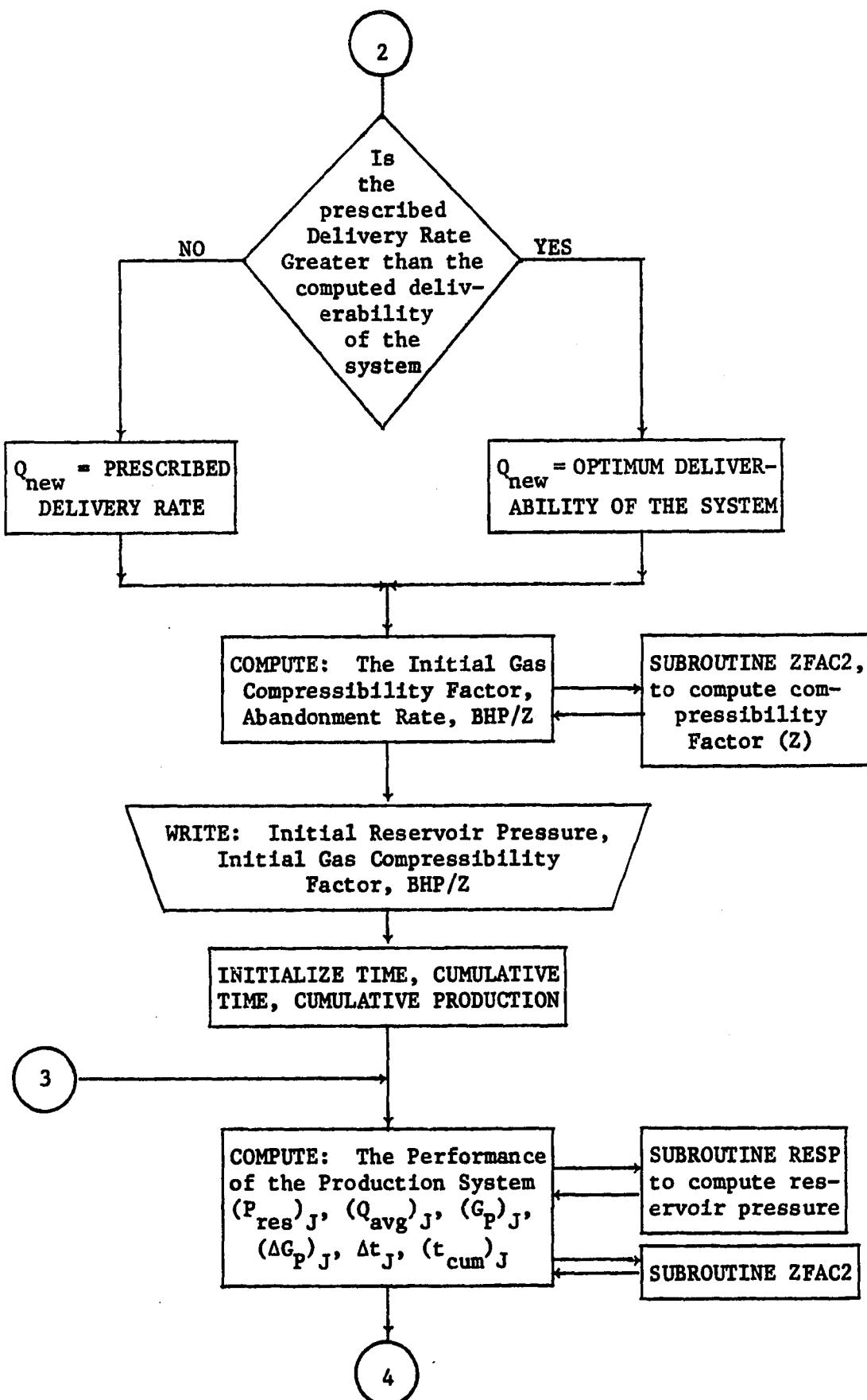
$$Q_o = \frac{Q'_o}{Q'_g} \times \frac{C}{C'} \times Q_g$$

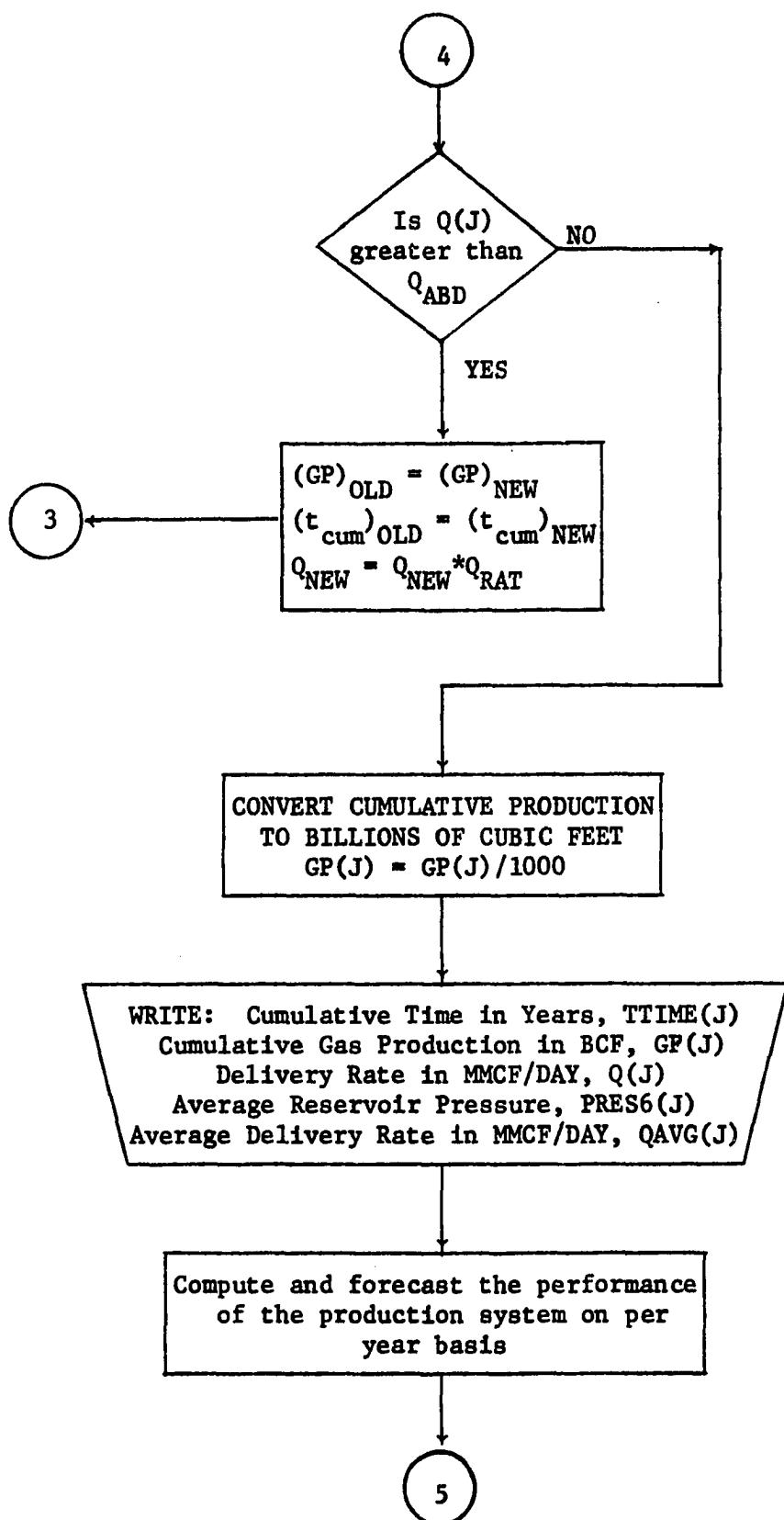
where C and C' are obtained from Figure 2, and $GOR' = Q'_g/Q'_o$ is prescribed in the computation.

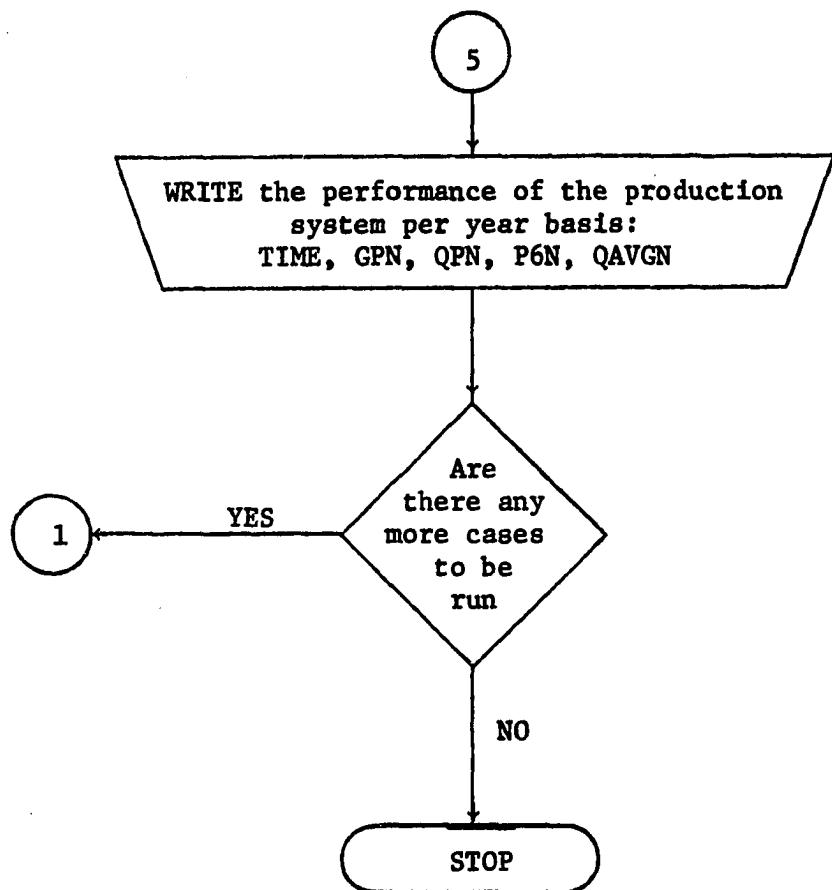
APPENDIX E

FLOW CHART OF THE COMPUTER PROGRAM FOR THE DRY GAS MODEL

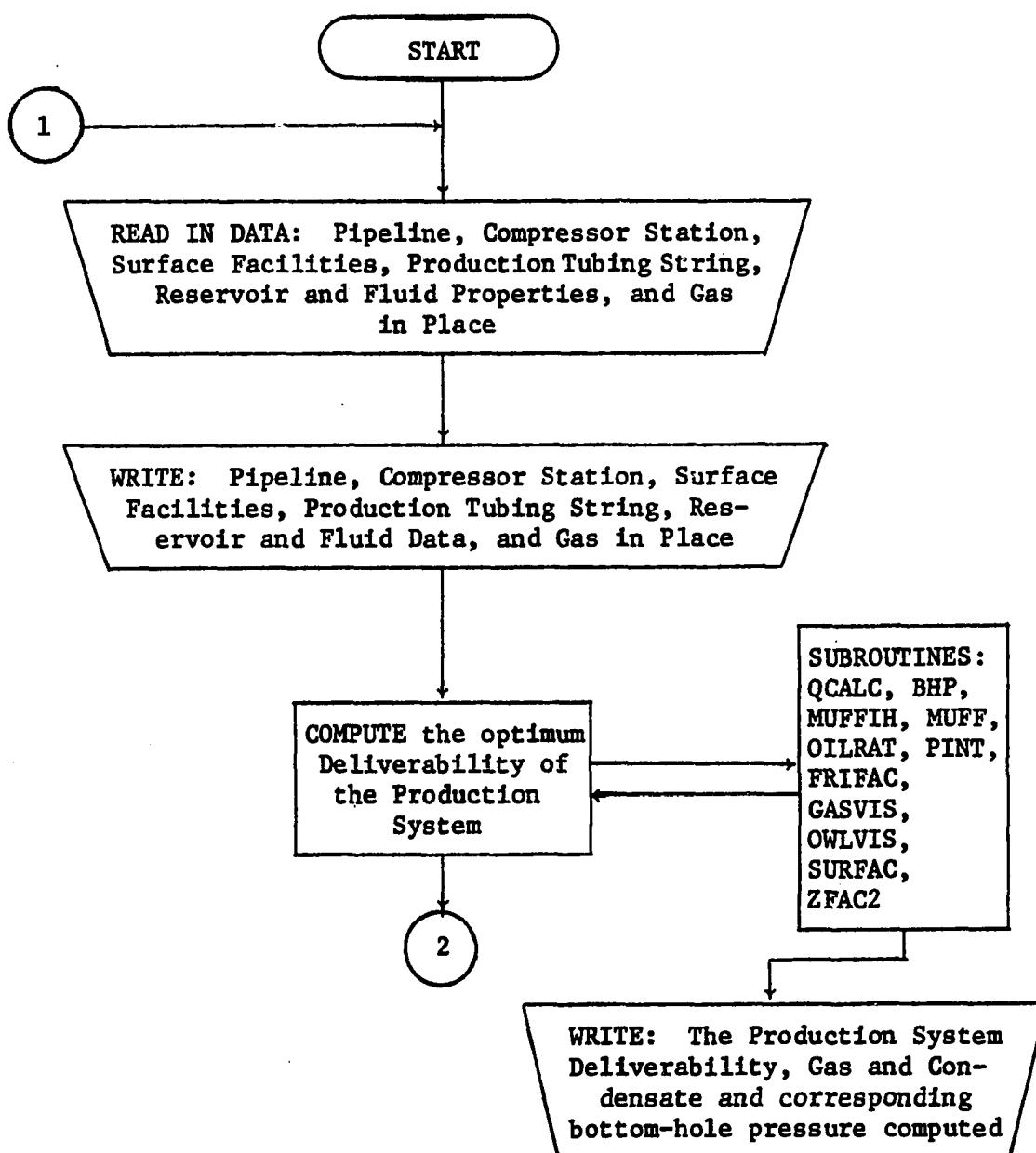


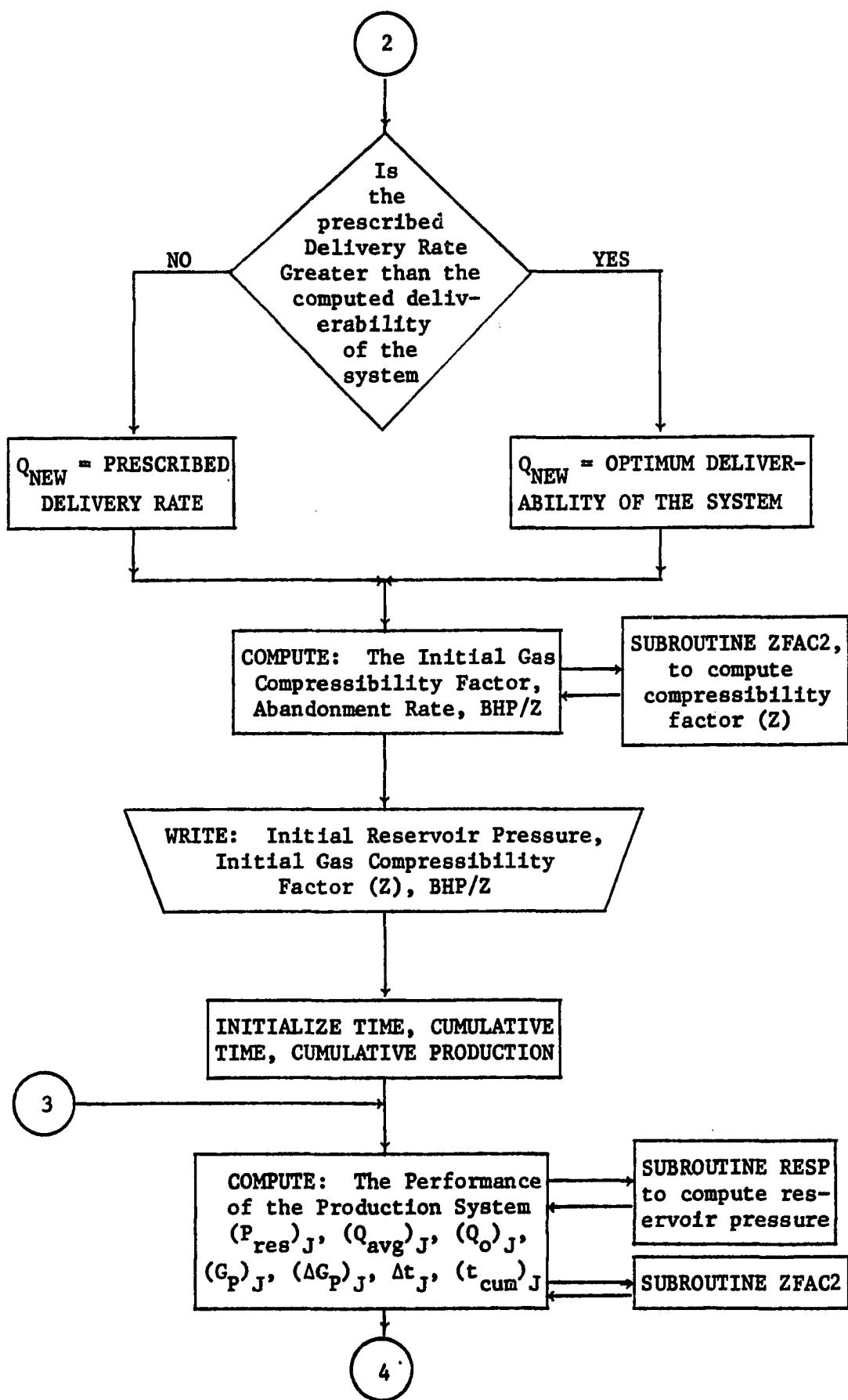


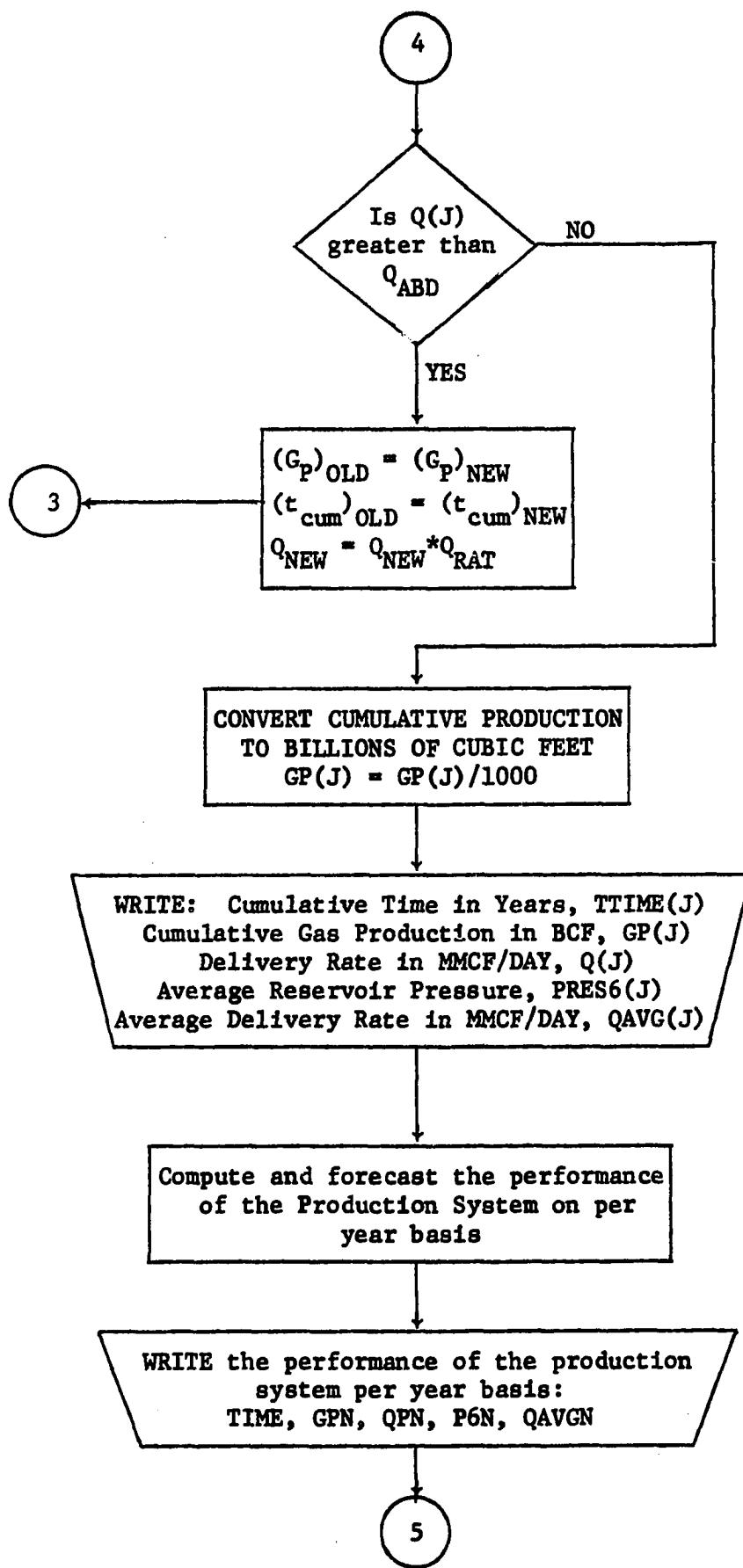


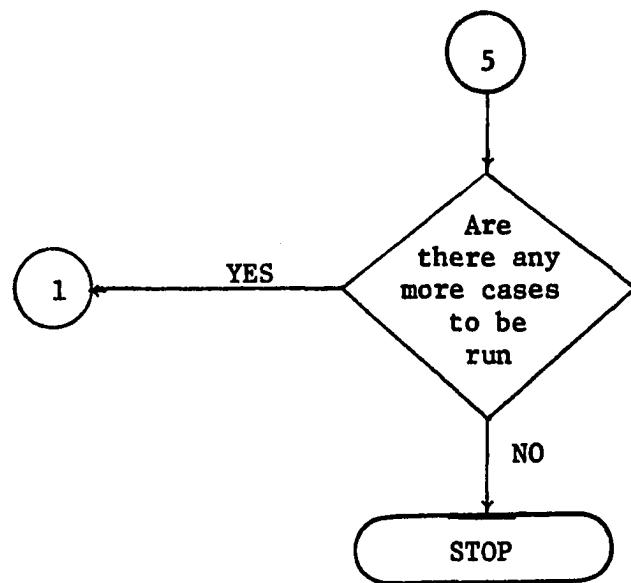


FLOW CHART OF THE COMPUTER PROGRAM FOR THE GAS-CONDENSATE SYSTEM









APPENDIX F

FORTRAN IV G LEVEL 21 MAIN DATE # 75151
 UNIVERSITY OF OKLAHOMA 1975 PH.D. RESEARCH
 INTEGRATED PERFORMANCE OF FLOW BEHAVIOR IN DRY GAS RESERVOIR,
 CLOUDINES FLOW BEHAVIOR OF RESERVOIR, FLOW STRING PROCESSING EQUIPMENT,
 GATHERING SYSTEM, COMPRESSION FACILITIES, TRUNKLINE.
 1) CALCULATIONS
 2) FLOW THROUGH THE RESERVOIR
 3) FLOW THROUGH THE PRODUCING STRING
 4) THROUGH THE SURFACE FACILITIES INCLUDING THE GATHERING
 SYSTEM, INTEGRATING EQUIPMENT AND GAS TREATMENT EQUIPMENT
 5) FLOW THROUGH THE COMPRESSOR STATION
 6) FLOW THROUGH THE PIPELINE TO THE SALES POINT
 7) MAXIMUM DELIVERABILITY OF A GAS PRODUCTION SYSTEM TO SALES CONNECTION
 8) PIPELINE IS EVALUATED
 P1 IS THE RELATIVE PRESSURE IN PSIA
 T1 IS THE PIPELINE TEMP. IN DEG. RANKIN
 D1 IS THE PIPELINE DIAM. IN INCHES
 X1 IS THE PIPELINE LENGTH IN MILES
 E1 IS THE PIPELINE EFFICIENCY FACTOR UNITLESS
 P0 IS THE BASE PRESSURE IN PSIA
 T0 IS THE BASE TEMP. IN DEG. RANKIN
 HP IS THE HORSE POWER IN HP
 RK IS THE COMPRESSION RATIO UNITLESS
 TS IS THE SUCTION TEMP. IN DEG. RANKIN
 E2 IS THE COMPRESSOR EFFICIENCY FACTOR UNITLESS
 Z3 IS THE Z-FACTOR AT COMPRESSOR SUCTION UNITLESS
 CS IS THE CONSTANT IN GATHERING LINE SYSTEM SCF/PSIA
 T4 IS THE AVERAGE TUBING TEMP. IN DEG. RANKIN
 Z4 IS THE Z-FACTOR IN THE TUBING UNITLESS
 D4 IS THE TUBING DIAM. IN INCHES
 X6 IS THE TUBING LENGTH IN FT.
 G IS THE GAS GRAVITY UNITLESS
 VIS4 IS THE GAS VISCOSITY IN CP.
 KELLA IS THE NM. OF THE FLLS UNITLESS
 P6 IS THE RESERVOIR PRESSURE IN PSIA
 T6 IS THE RESERVOIR TEMP. IN DEG. RANKIN
 T8 IS THE SLOPES OF OPEN FLOW CURVE IN MMCF/DAY/PSIA
 CS IS THE SLOPES OF OPEN FLOW CURVE IN MMCF/DAY/PSIA

FORTRAN IV G LEVEL 21 MAIN DATE 8 75151 11/26/43
 0039 CALL ZFAC0(A) FLOW1000
 0040 CALL ZFAC1(Z1,P6,T6,PC,TC,PR,86,A) FLOW1010
 0041 GSAVRIN FLOW1020
 0042 CALL DCALC(P1,11,01,X1,E1,P0,10,MP,RX,T3,T2,Z3,C3,14,24,04,FL0W1030
 1X4,6,V134,WEELS,P6,RN,CS,0N,PSL3,PSRS,PS,01)
 UXAN,
 IF (GSAVE .GT. UN) GO TO 77 FLOW1050
 UNGSAVE FLOW1060
 UNGSAVE FLOW1070
 IF (GSAVE .GT. UN) GO TO 77 FLOW1080
 UNGSAVE FLOW1090
 GO TO 88 FLOW1100
 0047 77 FLOW1110
 0048 0050 FLOW1120
 0049 QNSOK*QAT
 0051 R8 FLOW1130
 0052 QARD80,1,0X
 PIS86
 CALL ZFAC2(Z1,PI,T6,PC,TC,PR,86) FLOW1140
 PRZIAP123
 *R1TF(6,170)PI,Z1,PO21 FLOW1150
 0054 FLOW1160
 0055 FLOW1170
 0056 FLOW1180
 370 P14MMAT(SX, KSEWVNIR PRESS =',PRQ,2,5X,'Z1,PR10,6,5X,'/Z1,F12,2FL0M1180
 1,/) FLOW1190
 0057 0058 FLOW1200
 0059 0060 FLOW1210
 0061 0062 FLOW1220
 0063 DU 16,J82,100 FLOW1230
 0064 WRITE(J8P6,GP0,ON
 0065 GP(1)*GPD
 0066 TTIME(1)ATO
 0067 U1340N
 0068 PRFS6(J)GP6
 0069 DU 16,J82,100
 0070 WRITE(6,370) GPU,ON
 0071 FORMAT(SX,'HE3 PRESS(J),Z6(J),PO21'
 0072 F14,2,5X,'GAS RATE',F12,2,/,
 0073 X4,G,V134,*EELS,P6,RN,CS,0N,PSL3,PSRS,PS,01)
 0074 PRES6(J)GP6
 CALL ZFAC2(ZZ,PE,T6,PC,TC,PR,86)
 Z6(J)82Z
 PRZAPR36(J)/Z6(J)
 F81,*PO21,PO21
 GP(J)*FAG1
 WRITE(6,390) PRESS(J),Z6(J),PO21'
 F10,4,5X,'/Z1,PR10,2,5X,
 1,5X,'/Z1,PR10,2,5X,
 0(J)*SON
 0075 DAVG(J)*GART(QnABD)
 0076 DTIMT(J)*AGP(J)*GPD)/(NAVG(J)*J55,) FLOW1400
 0077 TTME(J)*DTIME(J)+TD FLOW1410
 0078 RTIME(5,40)*C(J),DTIME(J),TTIME(J)
 0079 RTIME(5,40)*C(J),DTIME(J),TTIME(J)
 400 DTIMT(5X,' GAS RATE ',F12,2,5X,'INCREMENTAL TIME ',F
 110,2,5X,' CUM TIME ',F11,10,2,/,
 NSTEP8J
 WRITE(6,201) NSTEP8J
 IF(C(J),LT, NABD) GO TO 17 FLOW1420
 0081 WRITE(6,201) NSTEP8J
 0082 FLOW1430
 0083 GP(GP(J)) FLOW1440
 0084 4000N
 0085 FLOW1450
 FLOW1460
 FLOW1470
 FLOW1480
 FLOW1490
 FLOW1500
 FLOW1510

FORTRAN IV LEVEL 2

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B118

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FORTNIGHT IV G LEVEL 21      RESP          DATE = 75151      11/26/45
1X4,G,V154,-P,L,S,P6,HN,CS,DN,PSL9,PSR9,PS,Q1)   FLO=M2160
C1W4WV/(WKL41/006,L   FLO=M2170
Q0D91,L,F6   FLO=M2190
C1B415,R7@t14(0/P0)*=1.07881   FLO=M2200
V1C1W4S0,00067272V154   FLO=M2210
T1F1X1,L,F,0.,0.R,D1,LE,0.) GU TO 10   FLO=M2220
Y1=(X1+T1)*(=0.539W)*G6*(=0.4606)*D1*E,6162   FLO=M2230
GU TO 20   FLO=M2240
Y160.   FLO=M2250
10   FLO=M2260
0010   CONTINUE   FLO=M2270
0011   IF(L1.LE.0.) GO TO 30   FLO=M2280
0012   IF(C1,LE,0.) GO TO 30   FLO=M2290
0013   Y250,(C1+V1)   FLO=M2300
0014   GU TO 40   FLO=M2310
0015   30   Y250,   FLO=M2320
0016   40   CONTINUE   FLO=M2330
0017   P2=SQT(P1+P1*Y2**1.054)   FLO=M2340
0018   M1=F1*(A,50)*P1,P2   FLO=M2350
0019   50   FORMAT(1/,5X,'DELIVERY PRESS P181,F10,2,5X,'DISCHARGE PRES P281
1,F10,2,/)   FLO=M2360
     RRKA7,RK,(RK=1,)   FLO=M2370
0020   IF(RRKA7,LT,0) GO TO 60   FLO=M2380
0021   Y3AWP/(0.0H531E-0RRKA7*0.73)+1.   FLO=M2390
0022   M1=F1*(A,50)*P1,P2   FLO=M2400
0023   IF(V3,LE,0.) GO TO 60   FLO=M2410
0024   P3SP2*Y3*(A(RRKA7/23))   FLO=M2420
     GU TO 70   FLO=M2430
0025   60   P3SP2*3.   FLO=M2440
0026   70   CONTINUE   FLO=M2450
0027   60   P3SP2*3.   FLO=M2460
0028   P1HWA15X,'SUCTION PRES P381,F10,2,/)   FLO=M2470
0029   60   P1HWA15X,'0.) GO TO 90   FLO=M2480
0030   IF(C3,LE,0.) GO TO 90   FLO=M2490
0031   P1HWA15X(P3*(P3+((0/C3)**2))   FLO=M2500
0032   GU TO 100   FLO=M2510
0033   P1HWA15X(P3+3.   FLO=M2520
0034   90   P1HWA15X(P3+3.   FLO=M2530
0035   100   CONTINUE   FLO=M2540
0036   110   P1HWA15X,1.'WELL HEAD PRES P481,F10,2,/)   FLO=M2550
0037   110   380*(0.375*GnX4/(T4*Z4))   FLO=M2560
0038   Y483/(T4*Z4)=1.,)   FLO=M2570
0039   Q180/VELL9   FLO=M2580
0040   QGG80/1.E6   FLO=M2590
0041   96686   FLO=M2600
0042   F80,0.009*(0.1*G/(V1C0N4*1,E6))**(-0.056)   FLO=M2610
0043   "RTTF(6,12)01   FLO=M2620
0044   0044   FORMAT(5X,'GAS RATE Q181,1,E15,7,/)   FLO=M2630
0045   F84*(T4/12),J,W(*0.056)   FLO=M2640
0046   Y58D4e5b/(G*2W*T4*F4*XQ)   FLO=M2650
0047   Y6a((1/2,FS)**2/(T4*Y5))   FLO=M2700
0048   P5L988,RT(PU*PU*EXP(S)*Y6)   FLO=M2710
     P1HWA15X(P5L98401)/(C581,E6))**(-1./RW)
0049   "RTTF(6,13)01   FLO=M2660
0050   WRITE(6,130)P5L9,P6   FLO=M2690
0051   FORMAT(5X,'FLWING RMP P5L981,F10,2,5X,'RES PRES P681,F10,2,/)   FLO=M2700
0052   RETURN   FLO=M2710
0053   END

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FORTAN IV G LEVEL 21          QCALC          DATE = 75151      11/26/45
                                SUBROUTINE QCALC(P1,T1,D1,X1,E1,P0,T0,MP,RK,T3,E2,Z3,C3,T4,Z4,D4,
                                1X4,G,VIS4,WELL3,P6,RN,CS,QN,PSL3,PSR3,P5,Q1)
                                COMMON/OKLA1/OGG,L
                                LS1
                                ITNU00
                                0004
                                0005
                                0006
                                DP1=SPB
                                50 ITNU001
                                CALL RMP(P1,T1,D1,X1,E1,P0,T0,MP,RK,T3,E2,Z3,C3,T4,Z4,D4,
                                1X4,G,VIS4,WELL3,P6,RN,CS,QN,PSL3,PSR3,P5,Q1)
                                SAW=SPB*(G1/(C5*1.06))**(1.0/RN)
                                IF (SAM.LE.0.2) GO TO 500
                                PSK=PSURT(PB*PB*(D1/(C5*1.06))**(1.0/RN))
                                G1=TR/400
                                500 SAW=SAW*(SAM)
                                PSK=PSK*(SAM)
                                400 CONTINUE
                                WRITE(6,201) ITNU0
                                WRITE(6,202) QN,PSL3,PSR3
                                IF (ITNU0.GT.100) GO TO 75
                                DP1=SPB*PSL3
                                IF (PSK*(DP1)=1.0) QQ=RN*RN
                                BN=0.1444*(DP1*(RN*G1))/(DP1*DPN)
                                QDQN
                                DYNARS(GSTAR)
                                DP1=SPB*
                                GO TO 50
                                75 WRITE(6,204)
                                201 FUM=15X,24WTHE SOLUTION AFTER TRIAL,15,16W
                                18 43 FOLLOW$//
                                202 FORMAT(15X,24WTHE GAS FLOW RATE IS F10.3,12W,
                                1 BMP CALCULATED FROM THE LINE SIDE IS,F10.3,W   PSIA,/5X,40WTHE FLOW3020
                                2 BMP CALCULATED FROM THE HS3. SIDE IS,F10.3,W   PSIA,/5X,40WTHE FLOW3030
                                204 FLOW AT (5X,29WHEN ITERATIONS HAVE BEEN MADE)
                                99 RETURN
                                END
                                0029
                                0030
                                0031

```


FORTRAN IV 6 LEVEL 21 SOLZ DATE # 75151 11/26/45

```

0001      SUBROUTINE SOLZ(P0Z,Z,P,1,PC,TC,PY,S,A)
0002          REAL A
0003          DIMENSION A(6,6),PY(6),S(6)
0004          P0Z=0.2
0005          I=1
0006          DO 100 I=1,6
0007              CALL ZFACT2(Z,I,P0,10,PC,TC,PY,S)
0008              P0Z=P0Z/2D0
0009              DELP0Z=DEL0Z-P0Z
0010              CALL ZFACT2(ZN,PN,TN,PC,TC,PY,S)
0011              P0Z=ZN/PN/ZN
0012              DELP0Z=DEL0Z-P0Z
0013              IF(ABS(DELPN)>L1,1) GO TO 88
0014              PSTARPN=(DELPN*(PN-P0))/DEL0Z
0015              P1=PN+PSTARPN
0016              PN=P1
0017              DELP0Z=DELPN
0018              I=I+1
0019              IF(I>N7,10) GO TO 90
0020              U=20
0021              R=17
0022              G1=17
0023              Z=ZN
0024              PPN=0
0025              FORMAT(5X,29HTEN ITERATIONS HAVE BEEN MADE)
0026              RETURN
          END
        ..

```

FORTRAN IV G LEVEL 21

ZFACO

DATE = 75151

11/26/45

```

SUBROUTINE ZFACO(A)
  DIMENSION A(6,6)
  REAL *8 A
  A(1,1)=2.145504
  A(2,1)=0.13176164D+1
  A(3,1)=0.21461142D+1
  A(4,1)=0.67143180D-3
  A(5,1)=0.42846483D+2
  A(6,1)=0.1656343D+2
  A(1,2)=0.3323524
  A(2,2)=0.1340314
  A(3,2)=0.66880061D-1
  A(4,2)=0.27178261D-1
  A(5,2)=0.84512691D+2
  A(6,2)=0.21520929D+2
  A(1,3)=0.1057271
  A(2,3)=0.50391654D+1
  A(3,3)=0.51920480D-2
  A(4,3)=0.10551356D+1
  A(5,3)=0.7318033D+2
  A(6,3)=0.24959635D+2
  A(1,4)=0.52180040D+1
  A(2,4)=0.44312146D+1
  A(3,4)=0.19320650D+1
  A(4,4)=0.58992560D+2
  A(5,4)=0.15366760D+2
  A(6,4)=0.28326809D+2
  A(1,5)=0.19713800D+1
  A(2,5)=0.26318354D+1
  A(3,5)=0.1926243D+1
  A(4,5)=0.1153139D+1
  A(5,5)=0.42910789D+2
  A(6,5)=0.13052526D+3
  A(1,6)=0.5309900D+2
  A(2,6)=0.69178310D+2
  A(3,6)=0.10890021D+1
  A(4,6)=0.9559390D+2
  A(5,6)=0.6011407D+2
  A(6,6)=0.3117517D+2
RETURN
END

```

FORTRAN IV G LEVEL 21

ZFACT

DATE # 75151

11/26/65

```

SUBROUTINE ZFACT1 (Z,P,T,PC,TC,PY,I3,A)
DIMENSION A(6,6),PY(6),S(6)
REAL *8A
VE(7)=((T/TC)+4.)/1.9
VSOPAV
VCUBEAVSGAV
VTRIUSVSOPAV
PY(1)=0.7071068
PY(2)=1.2247549
PY(3)=0.7905695*(3.*VSOPAV)
PY(4)=0.9354455*(5.*VCUBEAV)
PY(5)=0.26515*(55.*VT0430.*VSOPAV)
PY(6)=0.293151*(63.*VT0430.*VCUBEAV+15.*VSOPAV)
DN=20 I=1,J=0
S(I)=0.
ON IN J=1,6
S(I)=S(I)+A(J,I)*PY(J)
10 CONTINUE
20 C1=NTHIE
RETURN
END

```

FORTRAN IV G LEVEL 21

ZFACT2

DATE # 75151

11/26/65

```

SUBROUTINE ZFACT2 (Z,P,T,PC,TC,PY,I3)
DIMENSION PC(6,6),PY(6),S(6)
VE(2,*(P/PC))=15.,/14.8
VSOPAV
VCUBEAVSGAV
VTRIUSVSOPAV
PY(1)=0.7071068
PY(2)=1.2247549
PY(3)=0.7905695*(3.*VSOPAV)
PY(4)=0.9354455*(5.*VCUBEAV)
PY(5)=0.26515*(55.*VT0430.*VSOPAV)
PY(6)=0.293151*(63.*VT0430.*VCUBEAV+15.*VSOPAV)
DN=10 I=1,J=0
S(I)=0.
Z=Z+8*I*PY(I)
10 CONTINUE
RETURN
END

```

FORTRAN IV G LEVEL 21
PINT
DATE = 75151
11/26/65

```
0001      SUBROUTINE PINT(XA,YA,X,Y,M)
0002      DIMENSION X(20),Y(20)
0003      IP(XA,E0,X(1))GU 10 3
0004      DR 1 K#2,M
0005      IAK
0006      IF(XA.GT.X(M))GO TU 2
0007      IP(XA,E0,X(M))GU TU 4
0008      CONTINUE
0009      Y#((XA*X(I+1))/(X(I)*X(I+1)))*Y(I)=Y(I)+Y(I+1)
0010      GO TU 5
0011      Y=Y(I)
0012      GU TU 5
0013      Y=Y(I)
0014      RETURN
0015      END
```

FL0n4690
FL0n4700
FL0n4710
FL0n4720
FL0n4730
FL0n4740
FL0n4750
FL0n4760
FL0n4770
FL0n4780
FL0n4790
FL0n4800
FL0n4810
FL0n4820
FL0n4830

FORTRAN IV G LEVEL 21

MAIN

DATE # 75216

14/24/53

```

C***** FLOW 10
C***** AFLOW 20
C***** FLOW 30
C***** UNIVERSITY OF OKLAHOMA ***** FLOW 40
C***** 1975 ***** FLOW 50
C***** PH.D. RESEARCH ***** FLOW 60
C***** FLOW 70
C***** FLOW 80
C***** FLOW 90
C***** FLOW 100
C***** FLOW 110
C***** FLOW 120
C***** INTEGRATED PERFORMANCE OF FLOW BEHAVIOR IN GAS COND. RESERVOIR FLOW 130
C***** COMBINES FLOW BEHAVIOR OF RESERVOIR, FLOW STRING, PROCESSING EQUIPMENT FLOW 140
C***** GATHERING SYSTEM, COMPRESSION FACILITIES, TRUNKLINE. FLOW 150
C***** IT CALCULATES
C***** FLOW THROUGH THE RESERVOIR FLOW 160
C***** FLOW THROUGH THE PRODUCING STRING FLOW 170
C***** FLOW THROUGH THE SURFACE FACILITIES INCLUDING THE GATHERING FLOW 190
C***** SYSTEM, METERING EQUIPMENT AND GAS TREATMENT EQUIPMENT FLOW 200
C***** FLOW THROUGH THE COMPRESSOR SATURATION FLOW 210
C***** FLOW THROUGH PIPELINE TO THE SALES POINT FLOW 220
C***** FLOW 230
C***** FLOW 240
C***** MAXIMUM DELIVERABILITY OF A GAS PRODUCTIN SYSTEM TO SALES CONNECT FLOW 250
C***** UR PIPELINE IS EVALUATED FLOW 260
C***** FLOW 270
C***** FLOW 280
C***** FLOW 290
C***** FLOW 300
C***** P1 IS THE DELIVERY PRESSURE IN PSIA FLOW 310
C***** T1 IS THE PIPELINE TEMP. IN DEG. RANKIN FLOW 320
C***** D1 IS THE PIPELINE DIAM. IN INCHES FLOW 330
C***** X1 IS THE PIPELINE LENGTH IN MILES FLOW 340
C***** E1 IS THE PIPELINE EFFICIENCY FACTOR UNITLESS FLOW 350
C***** P0 IS THE BASE PRESSURE IN PSIA FLOW 360
C***** T0 IS THE BASE TEMP. IN DEG. RANKIN FLOW 370
C***** HP IS THE HORSE POWER IN HP FLOW 380
C***** RX IS THE COMPRESSION RATIO UNITLESS FLOW 390
C***** T3 IS THE SUCTION TEMP. IN DEG. RANKIN FLOW 400
C***** E2 IS THE COMPRESSUR EFFICIENCY FACTOR UNITLESS FLOW 410
C***** Z3 IS THE Z-FACTOR AT COMPRESSOR SUCTION UNITLESS FLOW 420
C***** C3 IS THE CONSTANT IN GATHERING LINE SYSTEM SCF/PSIA FLOW 430
C***** T0 IS THE AVERAGE TUBING TEMP. IN DEG. RANKIN FLOW 440
C***** Z4 IS THE Z-FACTOR IN THE TUBING UNITLESS FLOW 450
C***** D4 IS THE TURING DIAM. IN INCHES FLOW 460
C***** X4 IS THE TURING LENGTH IN FT. FLOW 470
C***** G IS THE GAS GRAVITY UNITLESS FLOW 480

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FORTRAN IV G LEVEL 21

MAIN

DATE # 75216

14/24/73

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0034      WRITE(6,320) MP ,          3
0035      320 FORMAT(//,5X,'H',          'F7.0', 'MP ', '8X, 'RK = ', 'B.2, 'UNITLESS ',
0036           'UX, ', 'I3 A ', 'F8.2, 'EG.R 1)
0037      325 FORMAT(//,5X,'E2.4 ', 'F0.2, 'UNITLESS ', 5X, '25 A ', 'F6.2, 'UNITLESS
0038           13 ')
0039      330 FORMAT(//,5X,'C3 A ', 'F10.1, '8CF/PSI ')
0040      335 FORMAT(//,5X,'T4 A ', 'F6.0, 'DEGR ', '6X, '24 A ', 'F6.2, 'UNITLESS
0041           '13X, ', 'D4 A ', 'F6.2, 'INCHES ', '4X, ', 'X4 A ', 'F7.0, 'FEET ')
0042      340 WRITE(6,340) GV134,WELL3
0043      340 FUNIT(//,5X,'G A ', 'F7.2, 'UNITLESS ', 3X, 'VISA A ', 'F6.4, 'CP ',
0044           '17X, ', 'WELL3 A ', 'F4.0, 'UNITLESS ')
0045      345 FORMAT(//,5X,'Pb/RHICb,Tb ', 'F7.0, 'PSIA ', '2X, 'RN = ', 'F4.2, 'UNITLESS
0046           '1,2X, ', 'C5 A ', 'F10.6, 'MMCF/DAY/PSIAM2N ', ', 'To ', 'F6.0, 'DEG')
0047      350 WRITE(6,350) GN
0048      350 FUNIT(//,5X,'ON A ', 'E20.4, 'MMSCF/DAY ')
0049      360 WRITE(b,360) GI
0050      360 FORMAT(//,5X,'INITIAL GAS INPLACE GI = ', 'E15.7, ')
0051      360 IF(IUEMUG.EQ.1) WRITE(6,360) HS,UVISL10,RHUIJL,000
0052      360 WRITE(6,360) FUNIT(//,5X,'RSZ ', 'F14.3, '5X, 'BSZ ', 'F14.3, '5X, 'LIUVISZ ',
0053           '1'RUWIL A ', 'F4.6, '5X, 'ULWHAT ', 'F14.6, ')
0054      360 IF(IDEBUG.EQ.1) WRITE(6,360) D2GAPP,V100,V210,SURFOL,FRCGI,QW
0055      360 DISDU/2.
0056      360 PSET38,A=19.85,IG=15,BB=64G
0057      360 IC=170,0+313,64G
0058      360 WRITE(6,360) WHAT,PC,TC
0059      360 CALL ZPAC0(A)
0060      360 USAVEQN
0061      360 CALL OCALC(P1,T1D1,X1,E1,P0,T0,MP,RK,T3,E2,I23,C3,T4,Z4,D4,
0062           IX4G,VIS4,WEL3,P6,RN,CG,GN,P5LS,P5RS,P5,01)
0063      360 OXON
0064      360 IF (OSAVE .GT. 0) GU TO 77
0065      360 GUOSAVE
0066      360 IF (OSAVE .GT. 0) GU TO 77
0067      360 UNOSAVE
0068      360 GO TU 68
0069      360 ONOSAVE
0070      360 GUADD=0.1 EQM
0071      360 P1=0
0072      360 CALL ZFACT2(Z1,P1,T0,PC,TC,PR,80)
0073      360 POZIAP1/21
0074      360 WAITE(6,1000)

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FORTRAN IV LEVEL 21

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DATE 3-25-96

14/44/54

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1000
FORMAT(1H1)
FORMAT(5X,' RESTRAINED PRESSURE =',F9.2,5X,' R/Z = ',F10.6,5X,' P/Z = ',F11.
1,/)
GPUAO.
TOMO.
GPT(J)AGPU
TIME(J)ATO
GP(J)AGPU
PRE56118P6
DU 16 JZ2,100
FORMAT(5X,'CUM GAS PROD =',F14.2,5X,'GAS RATE =',F12.2,')
CALL:      RESPP(J,T1,U1,X1,E1,P0,T0,H0,RK,J3,T2,Z3,C3,T4,Z4,DU
1XUG,VIS4,WELLS,P6,AN,C5,AN,P5L3,P5RS,P5,OL)
PRE56(J)MP6
CALL:      ZFAC2(ZZ,P6,T6,PC,TC,PR,86)
ZB(J)=ZZ
ZB(J)=ZB(J)/Z6(J)
GP(J)=FG1
GP(J)=GP(J)/Z6(J)
I_P = 1,F16.8,/
Q(J)=DN
UAVG(J)=SIN(J*Q(J))
DTIME(J)=GP(J)-GPU(J)/(UAVG(J)*365.)
TIME(J)=DTIME(J)+IN
WHITE(6,400)Q(J),DTIME(J),TIME(J)
FORMAT(' ', GAS RATE =',F12.2,5X,'INCREMENTAL TIME = ',F
10.2,5X,' , CUM TIME = ',F10.8,/)
NSTEPSJ
WRITE(b,201) NSTEPSJ
IF(O(J).LT. QABU) GU TO 17
GPUAGP(J)
QABU
TO5TIME(J)
UNSONARAT
CONTINUE
16  CONTINUE
17  DO 19 J=1,NSTEPS
18  GP(J)=GP(J)/1000.
19  CONTINUE
WRITE(6,40)
WHITE(b,42)  TIME(1),GP(1),Q(1),PRE56(1)
WRITE(b,41)  TIME(J),GP(J),Q(J),PRE56(J),AVG(J),JN2,NSTEPSJ
TIME(J)=TIME(NSTEPSJ)
TLM5ILIM
DPRINTNL.
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0119      TIME=0.
0120      PUMPRE36(1)
0121      UPUM(1)
0122      GPU0,
0123      WRITE(b,43) TIME,GPO,QPO,P60
0124      WRITE(b,42) TIME,GPO,QPO,P60
0125
0126      ISTART=2
0127      TIME=TIME+DTINR
0128      NSTANT=NSTANT
0129      DO 116 INSTART,NSTEPS
0130      IF(TIME .GT. TIME(I)) GO TO 116
0131      GPO=(GP(I)-GP(I-1))/(TIME(I)-TIME(I-1))*(TIME-TIME(I-1))
0132      1+GP(I-1)
0133      NAVGN=(GPN-GPO)/(DIPHIN=0.365)
0134      CALL 801Z(PUL,22,P6N,T6,PC,TC,PR,86,A)
0135      F=GPN=1000.*V61
0136      PUL=(1.0-F)*PNU1
0137      WRITE(b,41) TIME,GPN,OPN,P6N,NAVGN
0138      GP=GPN
0139      UPUMPN
0140      ISSTART
0141      IKTIM=1
0142      GO TO 117
0143      CONTINUE
0144
0145      ISTEP=1
0146      TIME=TIME(I)+((TIME(I)-TIME(I-1))/ALUG10(O(I))/Q(I-1))+ALUG10
0147      (QUBU/W(I-1))
0148      GPN=(GP(I)-GP(I-1))/(TIME(I)-TIME(I-1))*(TIME-TIME(I-1))
0149      1+GP(I-1)
0150      JAVEN=(GPN-GPO)/((TIME-TIME)=0.365)
0151      F=GPN=1000./GI
0152      PUZ=(1.0-F)*P023
0153      CALL 801Z(PUZ,ZZ,P6N,T6,PC,TC,PR,36,A)
0154      OPN=ABD
0155      WRITE(b,41) TIME,GPN,OPN,P6N,NAVGN
0156      GO TO 98
0157      FORMAT(1X,F5.2,0.4)
0158      FORMAT(1X,F5.2,0.4)
0159      FORMAT(1X,F5.2,0.4)
0160      FORMAT(1X,F5.2,0.4)
0161      FORMAT(1X,F5.2,0.4)
0162      FORMAT(1X,F5.2,0.4)
0163      FORMAT(1X,F5.2,0.4)
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0171      FORMAT(1X,F5.2,0.4)
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0194      FORMAT(1X,F5.2,0.4)
0195      FORMAT(1X,F5.2,0.4)
0196      FORMAT(1X,F5.2,0.4)
0197      FORMAT(1X,F5.2,0.4)
0198      FORMAT(1X,F5.2,0.4)
0199      FORMAT(1X,F5.2,0.4)
0200      FORMAT(1X,F5.2,0.4)
0201      FORMAT(1X,F5.2,0.4)
0202      STOP
0203      END

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FORTRAN IV LEVEL 21      UCALC      DATE 8 75216      14/24/53
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      SUBROUTINE UCALC(P1,T1,D1,X1,E1,P0,T0,MF,RR,T1,E2,Z3,C3,T4,Z4,D4,FLD*2450
     1X4,G,VIS4,WELL3,P6,RN,C5,Q4,P5L3,P5RS,P5,01)          FLDW2460
     COMMUN/DXLIN/L,UGG          FLDW2470
     COMMUN/DKLABK          FLDW2480
     K=0          FLDW2490
     L=0          FLDW2500
     FLW2510
     FLW2520
     FLW2530
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     FLW2750
     FLDW2760
     FLDW2770
     FLDW2780
     FLDW2790
     FLDW2800
     FLDW2810

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50 ITNU=ITNU+1

CALL RHP(P1,T1,D1,X1,E1,P0,T0,MF,RR,T1,E2,Z3,C3,T4,Z4,D4,

IX4,G,VIS4,WELL3,P6,RN,C5,Q4,P5L3,P5RS,P5,01)

SAMM(P6,P6*(01/(C5*1.E6)))*(1./RN))

IF(SAMM.LE.0.) GO TU 50U

P5RSURF(P6,P6*(Q1/(C5*1.E6)))*(1./RN))

GO TU 400

500 SAMM=AM(SAM)

P5NSORT(SAM)

400 CONTINUE

WRITE(6,201) ITNU

WRITE(6,202) DN,P5L3,P5RS

IF (ITNU .GT. 40.) GU 10 75

DPN=P5RS-P5L3

IF (ABS(DPN)>1.0) 99,80,80

80 USTAK=DN*(UN-QU)/(DPD-DPN)

DN=DN

UN=AN9(Q91AR)

DPD=DPN

GO TU 50

75 WRITE (6,204) 201 FORMAT(5X,20H,THE SOLUTION AFTER TRIAL,15,10H IS AS FULLD*9//)

202 FUKUHAT(5X,20H,THE GAS FLOW RATE IS,F10.5,12H MWSCF/DAV//5X,40H,THE FLDW2760
 1 RHP CALCULATED FROM THE LINE SIDE IS,F10.5,8H PSIA.,//5X,40H,THE FLDW2770
 2 RHP CALCULATED FROM THE RES. SIDE IS,F14.3,8H PSIA.,//) 203 FLDW2780
 204 FORMAT (5X,20H,THE ITERATIONS HAVE BEEN MADE) 205 FLDW2790
 99 RETURN 206 END 207 FLDW2800
 208 FLDW2810

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FURTHERN IV G LEVEL 21      BHP          DATE = 75216
0001   SUBROUTINE RHP(P1,T1,D1,X1,E1,P0,T0,MP,HH,C3,T4,ZS,C5,P5,Q1)    FLOW2820
        1X4,G,VISU,PULLS,P6,RN,C5,GN,P5LS,P5R3,P5,Q1)    FLUM2830
        COMMUN/DKLA/L,OGG    FLOW2840
        COMMUN/DKLA/R,K    FLOW2850
        GDN/R,K,E6    FLOW2860
        C1=0.55,87*(P1*(T0/P0)+1.07681    FLOW2870
        VICTHAT=0.0*D0.72*VISU    FLOW2880
        IF(X1*LE.0.0,MR.D1*LT.0.0) GU TO 180    FLOW2890
        Y1=(X1*T1)*R=(-0.5344)*C4*(-0.4606)*D1**2.6102    FLOW2910
        GU TU 190    FLOW2920
        180 Y1=0.
        190 CONTINUE    FLOW2930
        IF(X1*LE.0.) GO TO 140    FLOW2940
        IF(X1*LE.0.) GO TU 140    FLOW2950
        Y2=R/(C1*Y1)    FLOW2960
        GU TU 150    FLOW2970
        140 Y2=0.
        150 CONTINUE    FLOW2980
        P2=0.01*(P1+P1*Y2**1.854)    FLOW2990
        WRITE(6,202)    FLUM3010
        202 FWHAT(1M1)    FLUM3020
        WRITE(6,310)P1,P2    FLUM3030
        310 FWHAT(//,4X,'DELIVERY PRESS P1 =',F10.2,4X,'DISCHARGE PRESS P2'    FLUM3050
        1 3,F10.2,/)    FLUM3050
        RKRA=HK/(RK-1.)
        IF(FWHAT.LE.0.) GU TU 110    FLUM3060
        Y3BHP=(0.0H531E-0.0H5WKRAT+Q+T3)*41.
        IF(Y3,LE.0.) GO TU 110    FLUM3070
        P3=P2/73.3*(RKRA/723)    FLUM3080
        GU TU 130    FLUM3100
        P3=AP2*5    FLUM3120
        130 CONTINUE    FLUM3130
        WRITE(6,320)P3    FLUM3140
        320 FWHAT(//,5X,'SUCTION PRESS P3=',F10.2,/)    FLUM3150
        IF(C3,LE.0.) GO TU 160    FLUM3160
        PWSURV(P3+(LOG3)**2)    FLUM3170
        CALL MUFFIN(P3,T1,D1,X1,E1,P0,T0,Q,VICUNA,P4)    FLUM3180
        GU TU 170    FLUM3190
        P4=AP3*3    FLUM3200
        160 CONTINUE    FLUM3210
        170 WRITE(6,330) P4    FLUM3220
        330 FWHAT(//,5X,' THE WELL HEAD PRESS P4=',F10.2,/)    FLUM3230
        9=0.03756*X4/(T4*Z4)    FLUM3240
        Y4=S/(TEP(8)=1.0)    FLUM3250
        D12G=WELL9    FLUM3260
        F8U,U3U*(0.01*G/(VICUNA*1.E6))**(-0.065)    FLUM3270
        HH17L(6.340)Q1    FLUM3280
        0040 FWHAT(//,5X,' GAS RATE Q1=',E15,6,/)    FLUM3290
        0041 F8U*(U4/2.)**(-0.058)    FLUM3300
        0042 Y5D0**5*(G7Z7*T4*X4)    FLUM3310
        Y6=(U1/2.E5)**2*(Y4*Y5)    FLUM3320
        P5LS3S5U(P4*AP4*EXP(3)*Y6)    FLUM3330
        SGG3G    FLUM3340
        UGGD1/1.E6    FLUM3350
        CALL MUFF(P6,P4,T6,T0,X4,9GG,0GG,PSLS)    FLUM3360
        0043 WRITE(6,340)Q1    FLUM3370
        0044 F8U*(U4/2.)**(-0.058)    FLUM3380
        0045 P5LS3S5U(P4*AP4*EXP(3)*Y6)    FLUM3390
        SGG3G    FLUM3400
        0051 CALL MUFF(P6,P4,T6,T0,X4,9GG,0GG,PSLS)    FLUM3410
        0052 WRITE(6,350)P5LS    FLUM3420
        0053 FWHAT(//,5X,'P5LS=',F14,3,/)    FLUM3430
        0054 6000 FWHAT(//,5X,'P5LS=',F14,3,/)    FLUM3440
        0055 LBL1    FLUM3450
        0056 LBL1    FLUM3460
        0057 LBL1    FLUM3470
        0058 END    FLUM3480

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FORTRAN IV G LEVEL 21

NESP

DATE # 75216

14/24/55

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0001      SUBROUTINE RESPP(PI,T1,U1,A1,E1,P0,T0,HP,NK,IS,L2,Z5,C5,I4,Z4,D4,
1X4,G,V184,WELL,S,Ph,RN,C5,ON,PSL8,P5L9,PS,01)
0002      COMMUNIUKLA1/L,0GG
0003      UMH1A1,E6
0004      C1=455.874E1*(T0/PU)*+1.07881
0005      VICUN4=0.00672*1.94
0006      IF(X1.LE.0.01LE-0.) GO TO 10
0007      V1=(X1+1)*(=0.9394)*GG*(=0.4606)*D1*+2.6182
0008      GO TU 20
10      V1=0.
0009      20      CONTINUE
0010      IF(V1.LE.0.) GO TO 30
0011      IF((C1,LE.0.) GO TO 30
0012      U013      Y2=V1/(C1+V1)
0014      GO TU 40
30      V2=0.
0015      CONTINUE
0016      40      P2=80HT(P1+P1+Y2*+1.854)
0018      WRITE(6,50)P1,P2
0019      50      F10.2,'/')
1,F10.2,'/')
0020      HKA=HKA(RK=1.)
0021      U021      IF(HKA.LT.0.) GU TU 60
0022      U022      Y3AHP/(0.08531E-6*HKA*HKA*13)+1.
0023      I(F(KRAT,LE.0.) GU TU 60
0024      P3=P2/Y3A+(RKRAY/23)
0025      GO TU 70
0026      P3=P2*3.
60      P2=80HT(P1+P1+Y2*+1.854)
70      CONTINUE
0027      WRITE(6,80)P3
0028      F10.2,'/')
0029      FORMATT(5X,'SUCTION PRES P3',F10.2,'/')
0030      1F(C3,LE.0.) GO TU 90
0031      P4=80HT(P3+P3*(U/C3)*2)
0032      CALL      MUFIIM(P3),1,DI,X1,E1,P0,T0,0,VICON4,P4)
0033      GO TU 100
0034      P4=P3*3.
90      P4=P3*3.
100     CONTINUE
0035
0036      WRITE(6,110)P0
110      FORMATT(5X,'WELL HEAD PRES P4',F10.2,'/')
S=0.,0375=G0/((V4*Z4)
V48/(EXP(8)-1.0)
0038
0040
0041      WRITE(6,120)G1
0042      FORMATT(5X,'GAS RATE Q12 ''E18.7'',')
0043      F=0.3096*(0.65/(VICON4+1.E6))**(-0.065)
F=F*(D4/12.)**(-0.058)
0044
0045      Y5B=4.5/(G24*T4*F4)
0046      Y6=(U1/2.E5)*2/(V4*Y5)
0047      P1SSURR(P4*P4*EXP(9+Y6))
0048      SG=0
0049      UG=UG1/1.F6
0050      CALL      MUFF(P6,PU,16,10,X4,GGG,GGG,PSL9)
0051      WRITE(h,600)PSL9
6000      FORMAT(//,4X,'P5L9+G1/(C5+1.E6))**(1./RN),'
PESSUR(P5L9+P5L8+G1/(C5+1.E6))**(1./RN),
      K17T(h,130.PSL8/P6
0054      150      FORMATT(5X,'FLUXING HMP PSL8E',F10.2,5K,'RES PRES P02',F10.2,'/')
0055
0056      L0L1
0057      RETURN
0058      END.

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FORTRAN IV LEVEL 21

MUFFIN

14 / 24 / 53

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0001
0002 SUBROUTINE MUFFIN(P3,I,B,X1,E1,P0,T0,QM,V34,R4)
0003
0004 C(LIMIN/NKLA3)56
0005 COMMUN/NKLA4/PC,TC
0006 COMMUN/NKLA8/K
0007 COMMUN/NKLA9/R3,H0,VISL18,AMUDIL,QUU,G
0008 COMMUN/NKLA11,SURFOL,DEGAPI
0009 CALL ZFAC2(Z,RS,T1,HC,TC,PM,36)
0010 IF(K.GT.0) GO TO 700
0011 READ(S,601)M
0012 FURMAT(13)
0013 DU 602 151,M
0014 HEAD(5,20) X(),Y()
0015 FORMAT(2F14.5)
0016 CONTINUE
0017 DU 30 1M1,M
0018 X()=ALOG(X())
0019 Y()=ALOG(Y())
0020 CONTINUE
0021 DU 30 02R3*Z*T1/P3
0022 CONTINUE
0023 R=MUG3*2.7*NP3*G/(2*11)
0024 XUMX1
0025 API=DEGAPI
0026 EM0.0006
0027 PHI0.1414
0028 V1G3*EW184
0029 DIA .01/12.
0030 GAMAGC
0031 SURTSURFOL
0032 UGASGNVTE6
0033 HPGOGA000
0034 HPGOGA006
0035 UGASGU000*(H0+5.164/86400.
0036 UGASGU006*(HPGR8)*HG/86400.
0037 GAMAJI1.5/(131.5*API)
0038 AREASPH+DIA .02/.9.
0039 VSLOQVAREA
0040 VSLOQVAREA
0041 VSLOQVAREA
0042 VSLOQVAREA
0043 ALAMLS/VM
0044 VISMCAV1SL10*ALAML+VISGASR((=-ALAML)
0045 MUS00000H00TL
0046 wGAKKHUGASgGA
0047 wMASHUG
0048 XAB0.057*ASOR((wG00M)/(VISGASR*DIA .02/.9))
0049 XALUDG((MA))
0050 CALL, PRINT(XA,YA,PA,Y,M)
0051 UNDREP(YA)
0052 FFAUKW((w1/wM)4*1
0053 DPDXEFF*(wM*2/(64.35*DIA *AREASPH*2*RHM))
0054 DPDXEFFP/104.
0055 P2AD0P000000
0056 P4SP2*P3
0057 KAK*1
0058 RETURN
0059 END

```

```

0001      SUBROUTINE MUFF(PB,PS,TK,TOTALN,GGAS,O5,PSLS)
0002
0003      A1=(120),A2=(120),A3=(120),D1=(120),D2=(120)
0004      COMMUN/NKLA1/L,KXXX
0005      COMMUN/NKLA3/Sb
0006      COMMUN/NKLA1/SURFDEGAPI
0007      COMMUN/NKLA2/V100,V200,FRCG1,0M
0008      COMMUN/NKLA5/DI
0009      COMMUN/NKLA6/IDEBUG
0010      REAL,NU,A
0011      IFLC,G? ,0) GO TO 2221
0012      READ(S,31)PFNL,FFNL,PPK,FPK,PINT,FINT,FPR,PPR,QQP,QGP
0013      IF(TOTHUG.GT.1) GO TO 2221
0014      WRITE(6,35)PFNL,FFNL,PPK,FPK,PINT,FINT,FPR,PPR,QQP,QGP
0015      FORMAT(10FA,3)
35      FORMAT(10F? ,3)
0016      WRITE(b,500)RES,TSUMF
0017      WRITE(b,500)R5,SURFDEGAPI
0018      WRITE(6,500)Y100,Y210,FHCG1,0M
0019      FURWAT(6FL14,4)
0020      FURWAT(6FL14,4)
0021      WRITE(6,2500)PS
0022      FORMAT(5X,SURFACE PRESSURE,E14.3,/)
0023      CONTINUE
2221      CALL OILRAT(PB,PFNL,FFNL,PPK,FPK,PINT,FINT,FPR,PPR,QQP,QGP,QL)
0024      DQWL
0025      WRITE(6,29100
0026      FORMAT(5X,'NO=1,F10.3,5X,'HLS/DAV',/))
0027      L1=0
0028      GO TO 0029
0029      L21
0030
0031      T8T3UF
0032      G054G+1.E06
0033      G01L+1.5/(0.02GAP1+131.5)
0034      G0KNG/G01
0035      G0A0G/861000
0036      G0KAGDR+10.E6
0037      GRUMED1
0038      GRUMABRUM+12.
0039      EQUA=0.046/RQIM
0040      AREAUT=(3.141592/4)*01**2.
0041      TBAR(1)=TS
0042      GRAD=(TA-TS)/TOTALD
0043      XEIN=0.0*(.0128*DGAPI)
0044      YAO=0.0*(.00091*TBAR(1))
0045      KMAX/Y
0046      CMAX=1.0
0047      RQUL=&COEFF*GG19*((PS/10,1000)**(1./0.83))

```

FORTRAN IV G LEVEL 21 MUFF DATE # 75216

14/24/53

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0048      PHM3000.
0049      GM1R10.
0050      FLUM5090
0051      SUMW572.
0052      WOHW1.
0053      CALL ZFC0(A)
0054      CALL INEVIS(TBAR(1),WSULB,V100,V210,UVIS)
0055      DA(1)=U(0)
0056      DM(1)=U(0)
0057      UPSTATAD=0.005
0058      PD(1)=PS
0059      DDE30.
0060      AP(1)=PA
0061      K1=0.
0062      KAU.
0063      NELADD/2.
0064      DU 3 JZ2,60
0065      PWDSPD(J+1)+DP/2.
0066      CONTINUE
0067      PD(J)=AN(-1)+UP/2.
0068      TBAH(LJ)ANFL*GR4D+TS
0069      IC=170.+31.6*GGAS
0070      PCGH8H=19.85*GGAS+15.86*GGAS*GGAS
0071      TABAH=TRAN(J)+460.
0072      CALL ZFACT(Z1,PD(J),TABAR ,PC,TC,PY,Sb,A)
0073      CALL ZFACT(Z2,PU(J),TABAR ,PC,TC,PY,Sb)
0074      X1=10.+*(0.125*DEGPI)
0075      V1=10.**(.00091*TBAR(J))
0076      X2X1/Y1
0077      RSUL=CEFF*GGAS*((PD0/18.)*X1)*(1./.83)
0078      IF((GUH-RSUL)>6.,6.,7)
0079      GORESUL
6
7
0080      CONTINUE
0081      FACTUR1.25*14*AR(J)+RSUL*(GGAS/GDIL)+*,5
0082      IF(FACTUR1.LE.0.)GO TU 110
0083      BUR.000147*FACTUR1.175+0.972
0084      GO TU 111
0085      BUR.972
110
0086      CONTINUE
0087      CALL UWLVIS(PARR(J),WSUL ,V100,V210,UVIS)
0088      UL=6.49E-5*GUH
0089      UGAS=.27E-7*GUH*(GORRSOL)*(TBAR(J)+460.)/PD(J)
0090      GUH=UL*OG
0091      WL=DU*(4.05E-2*GGIL+8.65E-7*GGAS+RSOL)
0092      WG=8.85E-7*GUH*GGAS*(GORRSOL)
0093      WTUR=WL*WG
0094      RDU=WL/L
0095      IF(GUH)>6.,6.,9

```


FURTRAN IV G LEVEL 41 MUFF DATE # 75216 14/24/53

```

C
FLUM REGIME IS SLUG (LBBKG/01 & (LS)HVG
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      REYNULU1488.*VBLUT+DI+RUOIL/UVIS
      KK1=0,0
      KK2=KK1+1
      IF(KK1=13,)51,51,52
      51  RENUM=1488.*VBLUT+RUOIL+DI/UVIS
      IF(PEWNH=3000.)153,53,54
      IF(PEWNH=A0000.)155,55,56
      VBLAT=0.546*RUOIL/REYNULU*(32.2*DI)**.5
      DELT=VBLAT-VBLAT1
      IF(A=DELTA1=0.)157,57,58
      GJ TU 52
      58  VBLAT=VBLAT
      GO TU 59
      ALPH=0.251+8.74E-6*REYNULU*(32.2*DI)**.5
      VBLAT=ALPH/2.+((LPH**2.+13.5*ALV13/(RUOIL*UL**5))**.5
      DELT=VBLAT-VBLAT1
      IF(ARS(DELTA)=0.)160,60,61
      60  TU 52
      VBLAT=VBLAT
      GO TU 59
      61  TU 59
      VBLAT=VBLAT
      62  TU 52
      63  VBLAT=VBLAT
      GO TU 59
      64  TU 59
      52  IF(VTOT=10.)70,70,71
      70  CUN92=UVI3+1.
      DELTA90=0.01274*ALOG10((CUN91)/(DI **1.415)
      1.=2840.167*ALUG10(VTOT)+0.113*ALOG10(DI)
      CUN92=UVI3+1.
      DELTA90=0.0274*ALOG10((CUN92)/(DI **1.571))+.397*.63*ALOG10(DI)
      GO TU 72
      71  CUN92=UVI3+1.
      DELTA90=0.0274*ALOG10((CUN92)/(DI **1.571))+.397*.63*ALOG10(DI)
      GO TU 73
      72  GAMAB=0.065*VDT
      IF(DELTA=GAHA)74,74,75
      DELTAGAMA
      GO TU 78
      75  TU 78
      76  ROMARE=(WDT*RODIL*VBLAT*READY)/(TOT*VBLAT*READY)+DELTIA*RUOIL
      GAMAB=(VBLAT)/(VBLAT+VDT)*(1.-RUAR/RODIL)
      IF(DELTIA*GAMAB)79,79,78
      DELTAGAMA
      79  CONTINUE
      80  KUBAR=(WDT*RODIL*VBLAT*READY)/(TOT*VBLAT*READY)+DELTIA*RODIL
      81
      82
      83
      84
      85

```

FURTRAM IV C LEVEL 21 MUFF DATE 2 75216 14/24/53

FORTRAN IV G LEVEL 21

MUFF

DATE # 75216

14/24/53

```

      RUMIST=(1.-EG)*RUMIL+EG*RUG
      RUMARS=(XLIMM-VG)/(XLIMM-XLIMS)*RUMIST
      1+(VG-XLIMS)/(XLIMM-XLIMS)*RUMISLUG
      FNICLG=(XLIMM-VG)/(XLIMM-XLIMS)*FLSLUG
      1+(VG-XLIMS)/(XLIMM-XLIMS)*FLMIST
      CONTINUE
      CALCOP=144.*(DP *(1.-WTUT*QG/(4517.*AHEAU/*2.*PD(J))))/(RUBAR+
      ,FRICLG)
      DELNTPADH(J)=CALCOP
      DAMSABSD(J,0EP)
      IF(DAM=1.)95,95,96
      96  DM(J)=CH(J)-DEL66P/2.
      DEL66H(J)/2.*DA(J=1)
      LML1+1
      IF(LI=100)100,100,95
      DA(J)=CALCDP*DA(J=1)
      L100
      DELDD/2.*DA(J)
      AD(J)=UPAD(J=1)
      0236  IF(IDEBUG.FA.1)W4TF(6,2503)AD(J=1)
      2503 FORMAT(5X,'CUMULATIVE PRESSURE',*,F15.3)
      LAJ
      0239  IF(TOTALD)97,97,98
      97  IF(PM=AD(J))99,99,101
      98  IF(TOTALD=DA(J))99,99,101
      101  CONTINUE
      104  CONTINUE
      99  LL=L-1
      A0(LL)=ANULL
      IF(TUEPUG.FG.1)WHITE(6,2502)AD(LL)
      2502 FORMAT(//,5X,'BOTTOMMULE PRESSURE',F15.3//)
      PSLSMAD(LL)
      RETURN
      END
      FLUM7010
      FLUM7020
      FLUM7030
      FLUM7040
      FLUM7050
      FLUM7060
      FLUM7070
      FLUM7080
      FLUM7090
      FLUM7100
      FLUM7110
      FLUM7120
      FLUM7130
      FLUM7140
      FLUM7150
      FLUM7160
      FLUM7170
      FLUM7180
      FLUM7190
      FLUM7200
      FLUM7210
      FLUM7220
      FLUM7230
      FLUM7240
      FLUM7250
      FLUM7260
      FLUM7270
      FLUM7280
      FLUM7290
      FLUM7300
      FLUM7310
      FLUM7320
      FLUM7330
      FLUM7340

```

FORTNAN IV G LEVEL 21

ZFACO

DATE # 75216

14/24/53

	SUBROUTINE ZFACO(A)	DATE # 75216
0001	DIMENSION A(6,6)	14/24/53
0002	REAL*8 A	FLUM7350
0003	A(1,1)=2.1431504	FLUM7360
0004	A(1,1)*2.1431504	FLUM7370
0005	A(2,1)=0.83176184D-1	FLUM7380
0006	A(3,1)=-0.21467042D-1	FLUM7390
0007	A(4,1)=0.811403180D-3	FLUM7400
0008	A(5,1)=0.42866283D-2	FLUM7410
0009	A(6,1)=-0.6595343D-2	FLUM7420
0010	A(1,2)=0.33123524	FLUM7430
0011	A(2,2)=0.14036164	FLUM7440
0012	A(3,2)=0.668809510D-1	FLUM7450
0013	A(4,2)=0.27774261D-1	FLUM7460
0014	A(5,2)=0.88522910D-2	FLUM7470
0015	A(6,2)=0.21520924D-2	FLUM7480
0016	A(1,3)=0.1032871	FLUM7490
0017	A(2,3)=0.50393654D-1	FLUM7500
0018	A(3,3)=0.50921980D-2	FLUM7510
0019	A(4,3)=0.10551356D-1	FLUM7520
0020	A(5,3)=0.7381953D-2	FLUM7530
0021	A(6,3)=0.26959963D-2	FLUM7540
0022	A(1,4)=0.52184040D-1	FLUM7550
0023	A(2,4)=0.44312146D-1	FLUM7560
0024	A(3,4)=-0.1924465D-1	FLUM7570
0025	A(4,4)=0.58925516D-2	FLUM7580
0026	A(5,4)=0.15366676D-2	FLUM7590
0027	A(6,4)=0.28366809D-2	FLUM7600
0028	A(1,5)=0.14703980D-1	FLUM7610
0029	A(2,5)=-0.2633354D-1	FLUM7620
0030	A(3,5)=0.1626243D-1	FLUM7630
0031	A(4,5)=-0.1134539D-1	FLUM7640
0032	A(5,5)=0.42910890D-2	FLUM7650
0033	A(6,5)=0.8130526D-3	FLUM7660
0034	A(1,6)=0.53699900D-2	FLUM7670
0035	A(2,6)=0.8917831D-2	FLUM7680
0036	A(3,6)=0.1069882D-1	FLUM7690
0037	A(4,6)=0.959369D-2	FLUM7700
0038	A(5,6)=0.60114017D-2	FLUM7710
0039	A(6,6)=0.3117517D-2	FLUM7720
0040	RETURN	FLUM7730
0041	END	FLUM7740
		FLUM7750

FORTRAN IV G LEVEL 21

ZFACT1

DATE = 75216

1424/53

```

SUBROUTINE ZFACT1 (Z,P,T,PC,TC,PV,S,A)
DIMENSION A(6,6),PV(6),S(6)
REAL A(6,6)
Y=(2.*((T/TC)-4.))/1.9
YSQ=YY
YCUHESYSQ=Y
YTU4=YSQ*Y90
PY(1)=0.7071068
PY(2)=1.224745*Y
PY(3)=0.705695*(3.*YSQ-1.)
PY(4)=0.9154145*(5.*YCUBE-3.*Y)
PY(5)=0.265165*(35.*YT04-30.*YSQ+3.)
PY(6)=0.203151*(63.*YTU4*Y-70.*YCUBE+15.*Y)
DO 20 I=1,6
  S(I)=S((I)+A(J,I))*PV(J)
10 CONTINUE
20 CONTINUE
0019
0020
0021
      RETURN
END

```

FORTRAN IV G LEVEL 21

ZFACT2

DATE = 75216

1424/53

```

SUBROUTINE ZFACT2 (Z,P,T,PC,TC,PV,S)
DIMENSION Z(6,6),PV(6),S(6)
Y=(2.*((P/PC)-15.))/14.8
YSQ=YY
YCUHESYSQ=Y
YTU4=YSQ*Y90
PY(1)=0.7071068
PY(2)=1.224745*Y
PY(3)=0.705695*(3.*YSQ-1.)
PY(4)=0.9154145*(5.*YCUBE-3.*Y)
PY(5)=0.265165*(35.*YT04-30.*YSQ+3.)
PY(6)=0.203151*(63.*YTU4*Y-70.*YCUBE+15.*Y)
D1=0.1*1.6
Z=Z+3*(1.+PY(1))
10 CONTINUE
      RETURN
END

```

FORTRAN IV G LEVEL 21

14/24/53

DATE # 75216

UNLVIS

```

0001      SUBROUTINE UNLVIS(TEMPER,R90L,V1,V2,VISCOL)
0002      VISCOL=V1*((TEMPER/100.)**(1.345*ALOG((V2/V1))))
0003      IF(R90L>0.)1,2,2
0004      1      X=.996
0005      Y=.931
0006      GO TO 100
0007      2      IF(R90L>100.)3,4,4
0008      3      X=.820
0009      Y=.884
0010      GO TU 100
0011      4      IF(R90L>200.)5,6,6
0012      5      X=.7
0013      Y=.810
0014      GO TU 100
0015      6      IF(R90L>400.)7,8,8
0016      7      X=.545
0017      Y=.720
0018      GO TU 100
0019      8      IF(R90L>600.)9,10,10
0020      9      X=.499
0021      Y=.650
0022      GO TU 100
0023      10     IF(R90L>800.)11,12,12
0024      11     X=.375
0025      Y=.613
0026      GO TU 100
0027      12     IF(R90L>1000.)13,14,14
0028      13     X=.312
0029      Y=.575
0030      GO TU 100
0031      14     IF(R90L>1200.)15,16,16
0032      15     X=.273
0033      Y=.548
0034      GO TU 100
0035      16     IF(R90L>1400.)17,18,18
0036      17     X=.250
0037      Y=.522
0038      GO TU 100
0039      18     X=.230
0040      Y=.498
0041      190    VISCOL=VISCOL*exp
0042      IF(VISCOL-VISCOL)>0.,60,70
0043      60    VISCOL=VISCOL
0044      70    VISCOL=VISCOL
0045      RETURN
0046      END

```

FORTRAN IV G LEVEL 21 SOLZ DATE # 75216

```

0001      SUBROUTINE SOLZ(POL,Z,P,T,PC,TC,PY,S,A)
0002      REAL A
0003      DIMENSION A(6,6),PY(6),S(6)
0004      P0=PU
0005      CALL ZFACT(Z0,P0,T0,PC,TC,PY,S)
0006      PU=ZINH0/Z0
0007      DELP=PP0NZ0=POL
0008      PNA0,B5=PT0
0009      17 CALL ZFACT(ZN,PN,TN,PC,TC,PY,S)
0010      PN=ZN
0011      DELPN=ZN=POL2
0012      IF(ABS(DELPN).LT.1.) GO TO 88
0013      PSTAKBII=DELPD*(PN=PU)/(DELPN=DELP0)
0014      P0=PN
0015      PN=PSTAR
0016      DELP=DELPN
0017      ISI+1
0018      IF(I.GT. 10) GO TO 90
0019      GO TO 17
0020
0021      90 WRITE(6,91)
0022      88 Z=ZN
0023      PEPN
0024      FORMAT(5X,29H100 ITERATIONS HAVE BEEN MADE)
0025      RETURN
0026
END

```

FORTRAN IV G LEVEL 21 GASVIS DATE # 75216

```

SUBROUTINE GASVIS(PC,TC,TB,P,G,V18G)
DIMENSION PY(6),S(6)
COMMON/OKLA3/36
CALL ZFACT(Z,P,TB,PC,TC,PY,S6)
IF(P=.1.,.49,.49,.50)
V18G=0.047
GO TO 60
50   W=G*29
     AKS=(9.5,0.024W)*(TB*+1.5)/(209.+19.*W+TB)
     XA3.54*(986./TB)+.01*W
     Y=2.^4.-2^X
     QMUG=G*PI/1544.^(2*PI*1545.^#62.9)
     V18G=G*PI/1544.^(2*PI*1545.^#62.9)
     V18G=G*PI/1544.^(2*PI*1545.^#62.9)
     RETURN
60
END

```

14/24/53

```

FL0W8870
FL0W8820
FL0W8830
FL0W8840
FL0W8850
FL0W8860
FL0W8870
FL0W8880
FL0W8890
FL0W8900
FL0W8910
FL0W8920
FL0W8930
FL0W8940
FL0W8950
FL0W8960
FL0W8970
FL0W8980
FL0W8990
FL0W9000
FL0W9010

```

FORTRAN IV G LEVEL 21

PINT

DATE = 7/21/6

10/24/63

```

0001      SUBROUTINE PINT(XA,YA,X,Y,M)
0002      DIMENSION X(20),Y(20)
0003      IF(XA.LE.Y(1)), GO TO 3
0004      DU : K=2,M
0005      I=N
0006      IF(I.EQ.M) GO TO 4
0007      IF(XA.GT.Y(N)) AND X(NLT.X(K+1)) GO TO 2
0008      IF(XA.EQ.Y(N)) GO TO 4
0009      CONTINUE
0010      Y=(XA-X(1))/(X(1)-X(1+1))*(Y(1)-Y(1+1))+Y(1+1)
0011      GO TU 5
0012      Y=V(1)
0013      GU TO 5
0014      V=V(1)
0015      RETURN
0016
END

```

FORTRAN IV G LEVEL 21

SURFAC

DATE # 75216

14/24/53

```

SUBROUTINE SURFAC(SURF1,PRES,SURFEN)
0001      IF(PRES=550.)I=1,2
0002      SURF1=0.0025*PRES
0003      1
0004      GO TU 3
0005      2
0006      IF(PRES<1025.)I=4,5
0007      SURF1=0.0171*PRES
0008      GO TU 3
0009      5
0010      IF(PRES>1600.)I=6,7
0011      SURF1=2.4*0.025*PRES
0012      GO TU 3
0013      6
0014      IF(PRES=2500.)I=8,9
0015      SURF1=0.001*PRES
0016      GO TU 3
0017      8
0018      SURF1=0.2*0.016*PRES
0019      GO TU 3
0020      9
0021      IF(PRES=3910.)I=10,11,11
0022      SURF1=0.001*PRES
0023      GO TU 3
0024      11
0025      SURF1=SURF1/SURF/100.
0026      RETURN
0027
END

```

```

FORTMAN IV C LEVEL 21      PRIFAC      DATE 8/15/816      14/24/53
0001
0002
0003
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0011
0012
        SUMMURIVUE PRIFAC(MV,BUD,FRIC1,FACTOR)
        FRIC64/RV
        FRIC=64/RV
        FACTURE(1,((.74+2.4LUG(2,.4BUD+.18,68/(RV*FRIC1+.5)))))
        DELTA=FACTUR
        DELTAMHS(DELT)
        DELTAM=0.000560160.61
        IF(DELTA=0.000560160.61
        FRIC=(FACTOR*FRIC1)+2.
        GU TU 200
        IF(FACTOR=FRIC)62+63,63
        FACTUR=FRIC
        RETURN
        END

```