A METHOD OF ANALYSIS FOR NONLINEAR

DYNAMIC RESPONSE OF ARCHES

By

JERSON DUARTE GUIMARÃES

Engenheiro Civil Universidade Federal de Minas Gerais Belo Horizonte, MG, Brasil 1952

> Master of Science Duke University Durham, North Carolina 1967

Submitted to the Faculty of the Graduate College of the Oklahoma State University in partial fulfillment of the requirements for the Degree of DOCTOR OF PHILOSOPHY July, 1974

Thesis
19742
G963m
Cop. 2

andar Anna an Anna Anna an Anna an

and an an 1997 - Maria Andrea, an an Air an Air an 1997 - Angelan Angelan an Angelan Angelan 1997 - Angelan Angelan an Angelan

马拉拉 盐酸盐

STATE LEGENSE STATE LEGENSE STATE LEGENSE LEGENSE L MAR 13 1975 MAR 13 1975 MAR

A METHOD OF ANALMESTISDEFOOF MONALIMEETAREFOIF MONALIMEETAR FOR NONLINEAR DYNAMIC RESPONSENAMELOARESESDISTENAMELOARESESTIS

Thesis Approvednesis Approvednesis Approved:

Adviser INCS The Giberrates Contel agreed to Contel agreed wate College Dean of

902086 902086 902086

ACKNOWLEDGEMENTS

I wish to express my feelings of gratitude to the members of my Advisory Committee: to Dr. William P. Dawkins, major adviser and Chairman of the Committee, for his personal interest, guidance and invaluable suggestions; to Drs. Allen E. Kelly, John P. Lloyd and Donald E. Boyd, for their helpful assistance and advisement.

My special thanks to Ms. Suzanne Spears, for her help in the preparation of the drawings and to my fellow graduate students for their encouragement and consideration.

My deepest appreciation is extended to the School of Civil Engineering of the Oklahoma State University, and to the Universidade Federal de Goias, Goiania, Brazil, for providing the means which made this study possible.

I am most grateful to my devoted wife, Balbina, to my son, Gilson, and to my daughters, Gisele and Gislene, for the sacrifices, encouragement and patience given during the time spent achieving this goal. Certainly the constant inspiration and understanding of my entire family was a major factor for the successful completion of this work, and I hope that my effort corresponds to theirs.

a

iii

TABLE OF CONTENTS

Chapter Pag						
Ι.	INTRODUCTION	1				
II.	LITERATURE REVIEW	4				
III.	METHOD OF ANALYSIS	7				
	Mechanical Model	7 9 9 12 13 15 17 18 22				
IV.	COMPUTER PROGRAM	38				
۷.	VERIFICATION OF PROGRAM	51				
	Example Problem DTP1: Static Solution Only Example Problems DTP2: Linearly Elastic Response Example Problems DTP3: Inelastic Response	52 54 58				
VI.	APPLICATION OF PROGRAM	61				
	Problem DAP1: DYNARCH Application Problem 1 Problem DAP2: DYNARCH Application Problem 2	63 67				
VII.	SUMMARY, CONCLUSIONS AND RECOMMENDATIONS	69				
	Summary	69 70 71				
A SELE	CTED BIBLIOGRAPHY	72				
APPENDIX A - SOLUTION OF SIMULTANEOUS EQUATIONS						
APPENDIX B - LISTING OF PROGRAM DYNARCH						

Chapter												Page
APPENDIX	C -	PROGRAM	DYNARCH:	GUIDE	FOR DA	ATA I	NPUT	•••		••	••	121
APPENDIX	D -	PROGRAM SHEETS.	DYNARCH:	CODING	G LISTI	INGS	AND	PRINTO	ουπ • •		••	134

1

.

. .

•

LIST OF TABLES

Table						F	age							
D.1.	Nondimensional	ResultsProblem DTP2.1					•						•	163

LIST OF FIGURES

Figure	e Page	
3.1.	Mechanical Model	
3.2.	Cross Section	
3.3.	Typical Stress-Strain Curve	
3.4.	Strain and Stress Distributions	
3.5.	Freebody Diagrams of Bars and Joints	
3.6.	Geometric Configurations of Bars	
3,7.	Freebody Diagrams for Dynamic Solution	
3.8.	Types of Time Functions	
4.1.	Summary Flowchart of Program DYNARCH	
4.2.	Summary Flowchart for the Static Process	
4.3.	Summary Flowchart for the Dynamic Process	
5.1.	Static SolutionProblem DTP1	
5.2.	Data for Problem DTP2.1Linearly Elastic Dynamic Solution Only	
5.3.	Problem DTP2.2Linearly Elastic Dynamic Solution Only 57	
5.4.	Data for Problem DTP3.1Downward Sinusoidal Impulse 59	
6.1.	Data for DYNARCH Application Problems	
6.2.	Dynamic Loadings for DYNARCH Application Problems 64	
6.3.	Variation of Minimum Failure Load With Time of Duration 66	
C.1.	Typical Thrust-Moment Interaction Diagram.	

NOMENCLATURE

A _b , A _{sb}	transverse area of bottom reinforcement
A _i	area of a cross section at joint i
Ai	vector of coefficients in Equation (3.15)
Aj	area of segment j in a given cross section
As	transverse area of steel reinforcement
A _t , A _{st}	transverse area of top reinforcement
(AE) _i	extensional or axial stiffness of bar i
Bl	width of section at the top
B ₂	width of section at an intermediate depth
^B 3	width of section at the bottom
Bi	vector of coefficients in Equation (3.15)
C _i	vector of coefficients in Equation (3.15)
C _k	dimensionless constant for the k th mode of vibration
d _b	distance of centroid of bottom reinforcement to the top
	of the section
ਰ _i	depth of the centroid of the section at joint i
D _i	vector of coefficients in Equation (3.15)
d _j	distance of centroid of segment j to the top of the
·	section
d _t	distance of centroid of top reinforcement to the top of
	the section
E	Young's modulus of elasticity

viii

Ec	Young's modulus of elasticity for concrete or compression
Ej	Young's modulus of elasticity for segment j of a given
-	section, initial tangent or secant modulus
Es	Young's modulus of elasticity for steel
Et	Young's modulus of elasticity in tension
(EI) _i	flexural stiffness at joint i
f	rise of an arch
fr	modulus of rupture of concrete
fi xj	horizontal force at joint j of bar i
fi yj	vertical force at joint j of bar i
fi zj	moment at joint j of bar i
Fj	vector of static forces at joint j of bar i
F _m	maximum value or peak of time function
F(t)	dimensionless function of time
F _{xi}	vector of horizontal static and dynamic forces at joint i
F _{yi}	vector of vertical static and dynamic forces at joint i
h _k	distance of bottom of zone k to the top of the section
Н	horizontal reaction of an arch
i	an integer index, generally indicator of bar or joint
Ι	moment of inertia of the cross sectinn
Ι	impulse
j	an integer index or subscript
k	an interger index or subscript
L.	span of an arch or a beam. Length of a bar
Lo	span of a structure or length of a bar
L _{oi}	initial length of bar i
Li	length of bar i

•

.

ix

М	bending moment
M _c	bending moment at the crown of an arch
M.	bending moment at joint i
m	an integer index or subscript, number of joints
^m i	concentrated point mass at joint i
n	an integer index or subscript, number of bars
Ρ	concentrated applied load
Pi	point indicating location of joint i at the beginning of
	the dynamic process
p'	point indicating location of joint i after some elapsed
	time
р _о	uniform pressure or pressure pulse
^p cr	critical buckling pressure of an arch
p _{max}	maximum value of a pressure pulse
₽ _{mf}	minimum value of pressure pulse, capable of causing
	failure of an arch
p(α)	pressure pulse at angle α , radial
p(t,a)	magnitude of a radial pressure pulse at time t at angle α
Q _i	vector of applied forces at joint i, static
Q _{xi}	horizontal applied force at joint i, static
Q _{yi}	vertical applied force at joint i, static
Q _{zi}	applied moment at joint i, static
Q _{xdi}	horizontal dynamic applied load parameter
Q _y di	vertical dynamic applied load parameter
Q(t) Xdi	horizontal dynamic applied force at joint i, time i
Q(t) ydi	vertical dynamic applied force at joint i, time i
r .	radius of gyration of a section

X

R	radius of a circular arch
$R_{i}^{(j)}$	residue of equation i at trial j
S	stiffness matrix
S	total length of an arch
s ⁱ i.k	stiffness submatrix relating forces at joint j of bar i
0,7	to the displacement of joint k of bar i
t	elapsed time
t _r	time of rise of a time function
t _d	time of decay of a time function
T	smallest natural period of vibration
Tk	period of the k th mode of vibration
To	period of the "breathing" mode of vibration of a circular
	ring
T _i	thrust in bar i
U	vector of displacements
Ui	vector of displacements of joint i
u _{di} , u _{di}	horizontal dynamic displacement of joint i
u _{di} , u(t) di	horizontal velocity of point mass m _j
u _{di} , u(t) di	horizontal acceleration of point mass m _i
ü(r) di	residual horizontal acceleration at joint i
u _i	horizontal displacement of joint i
u _r	radial displacement
v _c	vertical displacement at the crown of an arch
v _{di} , v _{di}	vertical dynamic displacement of joint i
v _{di} , v(t) di	vertical velocity of point mass m _i
v _{di} , v(t)	vertical acceleration of point mass m _i
v(r) di	residual vertical acceleration at joint i

хi

v _i	vertical displacement of joint i
۷	vertical reaction of a structure
۷ _i	shear in bar i
X _i	abcissa of joint i, in global coordinate system
X _{oi}	initial abcissa of joint i
Υ _i	ordinate of joint i, in global coordinate system
Yoi	initial ordinate of joint i
Υ _j	distance from the centroid of segment j to the centroid
·	of a given section
z _k	a zone of the cross section (see Figure 3.2)
α	angle measuring range of applied dynamic load
αi	recursion coefficients in Equation (A.2) (Appendix A)
βi	recursion coefficients in Equation (A.2) (Appendix A)
∆Q _{xi}	unbalanced horizontal force at joint i
^Q _{yi}	unbalanced vertical force at joint i
Δt	time_interval
∆ ^θ i	relative rotation at joint i
ε	average axial strain in a bar
^e adi	average dynamic axial strain in bar i
^e ai	average combined static and dynamic strain in bar i
€asi	average static strain in bar i
εj	average strain in segment j of a section
μ	mass per unit length
φο	angle of opening of a circular arch
[¢] oi	initial average curvature at joint i
[¢] di	increment of curvature at joint i, due to dynamic condi-
	tions

xii

.

Φi	average curvature at joint i
[¢] si	average static curvature at joint i
[¢] ti	average curvature at joint i at time t
σ	average stress
σj	average stress in the segment j of a section
θi	slope of bar i
θoi	initial slope of bar i

•

CHAPTER I

INTRODUCTION

The purpose of this study is to develop a procedure for estimating the effects of nonlinear material characteristics on the behavior of arches under dynamic loads or combined static and dynamic loads.

Much work has been done in the investigation of the behavior of arches under transient loads, considering linear material properties and small displacement theory. Most of this work deals with determination of the lowest natural frequencies of vibration and corresponding mode shapes.

When effects of dynamic loads are combined with nonlinear properties of the materials, large displacements may take place at some points of the structure and the small deformation theory is unable to describe the variation of axial and flexural stiffnesses of the structure where large displacements occur.

The method developed herein takes into account material nonlinearity combined with geometric nonlinearity due to large displacements, by constantly revising the axial and flexural stiffness at selected sections of the structure, as deformations occur under transient loads.

'In view of the current importance of dynamic analysis for civil engineering structures, especially with regard to seismic and blast loading, the main application of this study is in the investigation of the

1

resistance of arches under the influence of transient loads, in the inelastic range of behavior of the materials.

A secondary, but no less important, application is to use the method here developed to investigate the effects of a high energy detonation in the vicinity of the structure, thus determining the efficient placement of a detonation to result in catastrophic collapse.

Also, if proper excitation loads are applied, natural modes of vibration, especially the first ones, can be obtained, and hence approximate natural frequencies can be determined for a variety of structures, such as circular, parabolic, elliptic, sinusoidal arches, with constant or variable cross sections of metallic materials or reinforced concrete.

The arches considered are geometrically determined by a given limited number of points forming the joints of a broken line in the plane of the arch, and may also have their axes described by any single-valued functions in the positive quadrant of the X-Y plane, starting at the origin of the coordinate system.

The material of the arch may be a composition of steel, aluminum or other metallic material, or reinforced concrete with top and/or bottom reinforcement, forming, in a specified way, rectangular, I or T sections, with uniform or variable area and mass per unit length.

The analysis is simplified by replacing the actual structure by a discrete framework, consisting of a finite number of bars, joints, concentrated point masses and springs, with properties based on the parameters of the original structure, such as geometry, boundary conditions and material properties.

The loads are applied at the joints as concentrated horizontal and/ or vertical loads, and may also be the result of uniform pressure,

2

linear or parabolic, vertical or horizontal, distributed loads. Static loads, if any, are considered only in the linear stage of the behavior of the materials of the structure.

A computer program is then developed in order to make possible the application of the method in the analysis of the replacement structure, with numerical evaluation of bending moments, thrusts, shears and deformations. Also collapse under specified failure criteria is determined.

CHAPTER II

LITERATURE REVIEW

Since Den Hartog (5), in 1928, presented the first study for the lowest frequency of vibration in extensional and inextensional modes for hinged-end and fixed-end circular arches, many investigators have been studying natural frequencies and mode shapes of vibration for circular arches and to a lesser degree noncircular geometries of arches having elastic curves of catenary, cycloidal and parabolic shapes have also been analyzed.

The papers by Wolf (10) and Veletsos et al. (9) give an extensive list of researchers who approached the problem of vibration of elastic arches or rings, using various procedures, such as the Rayleigh-Ritz method, or analytical solution of the differential equations of motion, or different techniques of numerical solution of the differential equations of motion.

Of particular interest is the work done by Eppink & Veletsos (6), where a replacement structure consisting of a series of bars, joints and concentrated point masses was used. The equations of motion were solved numerically by means of a step-by-step method of integration, due to Newmark (7). The material of the arch is linearly elastic, and the effects of shearing deformation and rotatory inertia were neglected. For circular hinged or fixed elastic arches, subjected to a transient normal pressure around the arch, with timewise variation approximated by

4

straight line segments, dynamic responses were obtained. By comparing the solutions with those by the modal method of analysis used by previous researchers, it was shown that excellent results could be obtained with a model structure of only twelve bars.

Wolf (10) used a similar replacement structure, a finite element model, and found the frequencies for the first six modes of vibration for hinged-hinged and fixed-fixed elastic circular arches in free vibration, using a direct-iterative eigensolution method. The effect of rotatory inertia is included, but transverse shear deformations are neglected. Tables of natural frequencies for circular arches of various slenderness ratios and angles of opening were presented.

Veletsos et al.(9) studied free vibration of elastic circular arches of uniform cross section and mass per unit length, either hinged or fixed at both ends. Numerical solutions were presented for the eight lowest natural frequencies of vibration. The associated mode shapes and strain energy distributions were also analyzed. It was shown that vibrational modes may be almost purely flexural, or almost purely extensional, or the extensional and flexural actions may be strongly coupled. Effects of rotatory inertia and shearing deformations were neglected.

Austin and Veletsos (1) extended the method used by Veletsos et al. to include the effects of rotatory inertia and shearing deformations. Numerical solutions were presented for the ten lowest natural frequencies of arches having an angle of opening of ninety degrees. It was shown that the effect of rotatory inertia may be appreciable only for small values of the slenderness ratio (length of the arch divided by the radius of gyration of the cross section), increasing with increasing order of frequency. The effect is most pronounced in the nearly horizontal segments, for which the arch is vibrating in a predominantly flexural mode, and least pronounced in the diagonal seqments, for which the arch is vibrating in a predominantly extensional mode. Rotatory inertia and shearing deformations were shown to have negligible effects in the primarily extensional modes of vibration.

Dawkins (3) used a similar lumped parameter model in the analysis of cylinderical tunnel liner-packing systems of reinforced concrete, sugjected to transient dynamic loading, where inelastic behavior of the constituents was permitted. The equations of motion for the model were solved by the Beta-Method of integration due to Newmark (7).

CHAPTER III

METHOD OF ANALYSIS

Mechanical Model

In order to define a method sufficiently general to handle a wide variety of parameters, it is necessary to represent the structure to be analyzed by a lumped parameter model, composed of straight bars and spring elements, which have force-deformation characteristics derived from the properties of the original members. Consequently, the actual system is represented by n bars and n+1 joints, as shown in Figure 3.1.

Theoretically, the behavior of the model becomes closer to the behavior of the actual structure as the number of bars gets larger. Each bar is considered massless, with its distributed mass concentrated as point masses at the ends of the bar. To every joint, station or node, and to every bar or element an identification number is assigned, from left to right, as indicated in Figure 3.1. Since there are n bars and n+1 joints, bar i is located between joints i and i+1. This model is similar to lumped parameter models used by Dawkins (3) and by Eppink and Veletsos (6).

Assumptions

In addition to the substitution of the mechanical model for the actual structure, the following general assumptions are made:

1. Plane sections remain plane;

7





ω

- 2. Loads and masses are concentrated at the joints;
- 3. Loads and displacements occur in the plane of the arch;
- 4. Linearly elastic behavior is assumed in order to determine the response of the arch to static loads, and then dynamic effects are superimposed on the initial static deformations;
- Effects of shearing deformation and rotatory inertia are neglected.

Cross Section Description

The general cross section shown in Figure 3.2 is used. It is defined at each joint of the idealized model structure and is divided into nine zones limited by lines parallel to the bases. Each zone may be of a different material, in order to allow a variety of composite sections. Each zone may be subdivided in a given number of fibers or segments, each segment with area A_j and with its centroid located at a distance d_j from the top of the section.

A top and/or bottom reinforcement with transverse areas A_t and A_b may also be provided at distances d_t and d_b , respectively, from the top of the section.

Stress-Strain Curves

A typical stress-strain curve for the materials of the cross section, including the reinforcement, is shown in Figure 3.3. The stressstrain curve is initially divided into ten regions, five for tension and five for compression. To each segment or fiber, as defined in the above description of the cross section, a stress-strain curve is initially assigned, according to the material of the segment, and the stress-strain



Figure 3.2. Cross Section



Figure 3.3. Typical Stress-Strain Curve

history of each segment is recorded, accounting for permanent sets, unloadings and reloadings. As the structure deforms, the stress-strain behavior of the material of the segment, in tension or compression, develops along the solid curve shown in Figure 3.3. If the direction of the strain is reversed, the stress-strain behavior is assumed to follow the dashed lines of Figure 3.3, parallel to the initial portion of the curve.

For the static solution, the material is assumed to be linearly elastic, with the stress-strain behavior defined in the initial portion of the curve, with a constant Young's modulus E_j equal to E_c in compression or E_t in tension.

For the dynamic solution, in the inelastic range of behavior of the materials, in order to take into account the modification of the transformed area and consequent shifting of the centroid of the cross section, as cracks and/or plastification are liable to occur, the modulus of elasticity E_j used is either the secant modulus, when the behavior of the material is expressed by the solid line of the stress-strain curve of Figure 3.3, or the initial tangent modulus, when the behavior of the material is expressed by the dashed lines of the stress-strain curve, corresponding to an unloading or reloading situation, with permanent set.

Centroid of the Section

The distance \overline{d}_i , measured from the centroid to the top of the cross section at joint i, is calculated considering the transformed area by the formula

$$\overline{d}_{i} = \left(\sum_{A_{j}} E_{j} A_{j} d_{j}\right) / \left(\sum_{A_{j}} E_{j} A_{j}\right)$$
(3.1)

12

with E_j , A_j and d_j as defined in the previous paragraphs and where the above summations are to be extended to all segments of the section A_i plus the top and bottom reinforcement, if any. The area of the material displaced by the steel is subtracted from the area of the segment in which the steel is placed.

Since E_j is constant in the static process, where elastic behavior and uncracked sections are assumed, the depth of the centroid $\overline{d_i}$ does not change, and the centroid is fixed throughout the entire static process. In the dynamic process, however, as plastification or cracks occur, E_j is no longer constant, since the secant modulus is used in the inelastic range, and the depth of the centroid varies, as calculated by Equation (3.1). This shifting of the centroid is described by J. Blaauwandraad (2), who also prescribed the use of the secant modulus in the analysis of nonlinear problems of reinforced concrete framed structures.

Strain and Stress Distribution

In a specified cross section i, if the location of the centroid \overline{d}_i , the average strain ε_{ai} and the curvature ϕ_i are known, a strain distribution can be determined, as shown in Figure 3.4.

The strain ε_{i} at the centroid of a segment can be calculated by

$$\varepsilon_{j} = \varepsilon_{ai} + \phi_{i} y_{j}$$
(3.2)

where $y_j = d_j - \overline{d}_j$ is the distance from the centroid of the segment to the centroid of the section. Tensile strains are positive. Curvatures are positive if they produce compressive strains at the top of the section.



Figure 3.4. Strain and Stress Distributions

Since ϵ_{ai} is defined in the bar and ϕ_i is defined at the joint, when Equation (3.2) is applied to a bar, ϕ_i is taken as the average between the curvatures of the adjacent joints, and when Equation (3.2) is applied to a joint, ϵ_{ai} is taken as the average between the strains in adjacent bars.

From the values of ε_j for each segment, a stress distribution can be determined from the stress-strain curve for that particular segment, where, given ε_j and the strain history, the secant modulus E_j and the stress σ_j can be obtained.

If an unloading situation has not yet occurred, the equation

$$\sigma_{j} = E_{j} \varepsilon_{j}$$
(3.3)

holds, not only in the linearly elastic behavior, where E_j is constant, but also in the nonlinear range, where E_j is variable.

Thrusts, Moments and Shears

The thrust in bar i can be calculated by the equation

$$T_{i} = \sum_{A_{j}} A_{j} \sigma_{j}$$
(3.4)

where A_i is the cross section at the midpoint of bar i.

The bending moment at joint i can be obtained by the equation

$$M_{i} = \sum_{A_{i}} A_{j} \sigma_{j} y_{j}$$
(3.5)

where A_i is the cross section at joint i.

Taking Equation (3.4) and substituting σ_j and ϵ_j for the values in Equations (3.3) and (3.2):

$$T_{i} = \sum_{A_{i}} A_{j} E_{j} \varepsilon_{j} = \sum_{A_{i}} A_{j} E_{j} (\varepsilon_{ai} + \phi_{i} y_{j})$$

$$T_{i} = \varepsilon_{ai} \sum_{A_{i}} A_{j} E_{j} + \phi_{i} \sum_{A_{i}} A_{j} E_{j} y_{j}.$$

The second term in the right hand side of the above equation vanishes, since

1

$$\sum_{A_{j}} A_{j} E_{j} y_{j} = 0$$

is the condition imposed to obtain the depth of the centroid calculated by Equation (3.1).

As a matter of fact, by substituting into the above equation y_j for $d_j - \overline{d}_i$, one obtains $\sum_{A_i} A_j E_j (d_j - \overline{d}_i) = 0$

from which

$$\overline{d}_{i} = \left(\sum_{A_{i}} A_{j} E_{j} d_{j}\right) / \left(\sum_{A_{i}} A_{j} E_{j}\right)$$

is Equation (3.1) itself.

Then, the value of T_i is given by

$$T_{i} = \varepsilon_{ai} \sum_{A_{i}}^{N} A_{j} E_{j}.$$
 (3.6)

Now, taking Equation (3.5) and substituting σ_j and ϵ_j for the values in Equations (3.3) and (3.2)

$$M_{i} = \sum_{A_{i}} A_{j} E_{j} (\epsilon_{ai} + \phi_{i} y_{j}) y_{j}$$

or

$$M_{i} = \epsilon_{ai} \sum_{A_{i}}^{\sum} A_{j} E_{j} y_{j} + \phi_{i} \sum_{A_{i}}^{\sum} A_{j} E_{j} y_{j}^{2};$$

and, since $\sum_{A_{j}} A_{j} E_{j} y_{j}$ must vanish, the bending moment is given by $M_{i} = \phi_{i} \sum_{A_{j}} A_{j} E_{j} y_{j}^{2}.$ (3.7)

Shear in bar i is calculated by

$$V_{i} = (M_{i+1} - M_{i})/L_{i}$$
(3.8)

where L_i is the length of bar i.

١

Extensional and Flexural Stiffnesses

From Equation (3.6), the average extensional or axial stiffness in bar i can be defined as

$$(AE)_{i} = \sum_{A_{i}} A_{j} E_{j}$$
,

and the thrust in bar i can be expressed as

$$T_{i} = \varepsilon_{ai} (AE)_{i} .$$
 (3.9)

From Equation (3.7), the average flexural stiffness at joint i can be defined by

$$(EI)_{i} = \sum_{A_{i}} A_{j} E_{j} y_{j}^{2},$$

and the bending moment at joint i can be expressed as

$$M_{i} = \phi_{i} (EI)_{i}$$
 (3.10)

Equations (3.4) and (3.5) can be used to determine the thrust and the bending moment at any stage of loading or behavior of the material, but Equations (3.9) and (3.10) can be used to advantage when no unloadings and reloadings are taking place or when only linearly elastic behavior is considered.

Static Solution

It is assumed that, under static loads, the deformations of the structure will be small and conventional matrix analysis can be employed. The bars or elements of the structure are considered straight, with constant cross section, although the cross section is allowed to vary along the structure.

Using the notation

$$F_{j}^{i} = F_{joint j}^{bar i} = \begin{cases} f_{xj}^{i} \\ f_{yj}^{i} \\ f_{zj}^{i} \end{cases}; \qquad U_{i} = U_{joint i} = \begin{cases} u_{i} \\ v_{i} \\ \Delta \theta_{i} \end{cases}$$

and considering bar i, between joints i and i+1, as shown in Figure 3.5, the following matrix equations can be written:

$$\begin{cases} F_{i}^{i} \\ ---- \\ F_{i+1}^{i} \end{cases} = \begin{bmatrix} S_{i,i}^{i} & S_{i,i+1}^{i} \\ ----- \\ S_{i+1,i}^{i} & S_{i+1,i+1}^{i} \end{bmatrix} \begin{cases} U_{i} \\ ---- \\ U_{i+1} \end{cases}$$
(3.11)

or symbolically F = SU, where S is the stiffness matrix for bar i, in the global coordinate system X-Y of Figure 3.5(a).

Equation (3.11) relates the forces at the ends of bar i to the respective displacements, through the stiffness coefficients of bar i. $S_{j,k}^{i} = S_{joint j,joint k}^{bar i}$ is a 3 x 3 submatrix, which relates the forces at joint j of bar i to the displacements of joint k of bar i.

In the calculation of the stiffness matrix S, the bar is considered of constant cross section. Since the cross section is permitted to vary along the structure, and the axial stiffness AE and the flexural stiffness



Figure 3.5. Freebody Diagrams of Bars and Joints

61

EI are defined at the joints, the average values of adjacent joints are considered in the determination of the stiffness matrix S for bar i:

$$(AE)_{bar i} = \frac{1}{2} [(AE)_{joint i} + (AE)_{joint i+1}];$$

$$(EI)_{bar i} = \frac{1}{2} [(EI)_{joint i} + (EI)_{joint i+1}].$$

For bar i-1:

$$\begin{cases} F_{i-1}^{i-1} \\ F_{i-1}^{i-1} \\ F_{i}^{i-1} \end{cases} = \begin{bmatrix} S_{i-1,i-1}^{i-1} & S_{i-1,i}^{i-1} \\ \cdots & S_{i-1,i-1}^{i-1} & S_{i,i}^{i-1} \end{bmatrix} \begin{cases} U_{i-1} \\ \cdots & U_{i-1} \\ U_{i-1} \\ U_{i-1} \\ U_{i-1} \end{cases}$$
(3.12)

For equilibrium of joint i:

$$F_{i}^{i-1} + F_{i}^{i} = Q_{i}$$
 (3.13)

where Q_i is the vector of applied forces at joint i:

$$Q_{i} = \begin{cases} Q_{xi} \\ Q_{yi} \\ Q_{zi} \end{cases} .$$

From Equation (3.12), the value of F_{i}^{i-1} can be expressed as a 3 x 1 submatrix:

$$F_{i}^{i-1} = S_{i,i-1}^{i-1} U_{i-1} + S_{i,i}^{i-1} U_{i}$$
.

From Equation (3.11), the value of F_i^i can also be expressed as the 3 x 1 submatrix:

$$F_{i}^{i} = S_{i,i}^{i} U_{i} + S_{i,i+1}^{i} U_{i+1}$$
.

Taking the above values into Equation (3.13),

$$S_{i,i-1}^{i-1} U_{i-1} + [S_{i,i}^{i-1} + S_{i,i}^{i}] U_{i} + S_{i,i+1}^{i} U_{i+1} = Q_{i}$$
,

which can be written in the form:

$$A_{i} U_{i-1} + B_{i} U_{i} + C_{i} U_{i+1} + D_{i} = 0.$$
 (3.14)

Since A_i , B_i , C_i depend on the stiffnesses of bars i-1 and i, and D_i depends on the applied forces at joint i, these coefficients can be obtained for every joint, given the geometry and physical properties of the bars and the applied loads.

In order to determine the displacements of the joints of the structure, the following system of linear simultaneous equations must be solved:

Joints	Equations
1	$B_1U_1 + C_1U_2 + D_1 = 0$
2	$A_2U_1 + B_2U_2 + C_2U_3 + D_2 = 0$
•	· · · · · · · · · · · · · · · · · · ·
i	$A_{i}U_{i-1} + B_{i}U_{i} + C_{i}U_{i+1} + D_{i} = 0$
•	• • • • • • • • • • • • • • • • • •
n	$A_{n}U_{n-1} + B_{n}U_{n} + C_{n}U_{n+1} + D_{n} = 0$
m = n+1	$A_{m}U_{n} + B_{m}U_{m} + D_{m} = 0$
	(3.15)

Using a well known variation of Gauss elimination procedure for solving simultaneous equations, shown in Appendix A, the displacements are determined and the values of forces at the ends of all bars can be found by Equation (3.11). Then, bending moments, shears and thrusts can be determined.

Strains and curvatures, due to the applied static conditions, can now be calculated, by taking thrusts and bending moments, axial and flexural stiffnesses into Equations (3.9) and (3.10) to obtain: Average static strain in bar i:

$$\varepsilon_{asi} = T_i / (AE)_i. \tag{3.16}$$

Average static curvature at joint i:

$$\phi_{si} = M_i / (EI)_i.$$
 (3.17)

Dynamic Solution

Initial Structural Configuration

At the beginning of the dynamic process, the structure is already deformed under the static loads, with all acting and resisting forces in equilibrium.

Every joint is defined by its coordinates with respect to a given Cartesian coordinate system X-Y, which include the displacements caused by the static loads.

Let P_{i-1} (X_{o(i-1)}, Y_{o(i-1)}), P_i (X_{oi}, Y_{oi}) and P_{i+1} (X_{o(i+1)}, Y_{o(i+1)}) be three consecutive joints defining two consecutive bars, Figure 3.6, at the beginning of the dynamic process.

The length of bar i, between joints P_i and P_{i+1} , can be expressed as

$$L_{oi} = \sqrt{(\chi_{o(i+1)} - \chi_{oi})^2 + (Y_{o(i+1)} - Y_{oi})^2}.$$

The initial slope of bar i is given by

$$\theta_{oi} = \operatorname{Arcsin}\left[\frac{Y_{o(i+1)} - Y_{oi}}{L_{oi}}\right].$$

The initial average curvature at joint i is

$$\phi_{oi} = \frac{\theta_{oi} - \theta_{o(i-1)}}{\frac{1}{2} [L_{oi} + L_{o(i+1)}]}$$


Configuration of the Structure at Time t

After some elapsed time from the moment of application of dynamic forces, the three consecutive joints will occupy positions P'_{i-1} , P'_i and P'_{i+1} , Figure 3.6, in view of the dynamic displacements $u_{d(i-1)}$, u_{di} , $u_{d(i+1)}$ in the horizontal direction and $v_{d(i-1)}$, v_{di} , $v_{d(i+1)}$ in the vertical direction.

Now the length of bar i is expressed as

$$L_{i} = \sqrt{[X_{(i+1)} + u_{d(i+1)} - X_{oi} - u_{di}]^{2} + [Y_{o(i+1)} + v_{d(i+1)} - Y_{oi} - v_{di}]^{2}}.$$

The slope of bar i at this time is

$$\theta_{i} = \operatorname{Arcsin}\left[\frac{\gamma_{o(i+1)} + \nu_{d(i+1)} - \gamma_{oi} - \nu_{di}}{L_{i}}\right].$$

The average curvature at joint i is

$$\phi_{ti} = \frac{\theta_i - \theta_{i-1}}{\frac{1}{2} \left[L_i + L_{i-1} \right]}$$

The dynamic average axial strain in bar i is

$$\varepsilon_{adi} = \frac{L_i - L_{oi}}{L_{oi}} = L_i/L_{oi} - 1.$$

The increment of curvature at joint i due to the applied dynamic conditions is

$$\phi_{di} = \phi_{ti} - \phi_{oi}$$

The total average strain and curvature are obtained by adding the contribution of the static process, from Equations (3.16) and (3.17), to the dynamic strain and curvature:

Strain and Stress Distribution at Time t

With the values of average strain and curvature, ε_{ai} and ϕ_i , a distribution of strains and stresses in the cross section, similar to those of Figure 3.4, necessary for calculation of internal forces, can be determined. One difficulty arises because, since cracking and/or plastification may have occurred, the location of the centroid of the transformed area is not known at time t.

The following iterative procedure is then used to determine an acceptable location of the centroid and consequent distribution of strains and stresses:

- A position of the centroid of the transformed cross section is assumed, either as the original one used in the static solution or one obtained in a previous step;
- 2. A trial distribution of strains can be determined for the section, using Equation (3.2), and finding the strains ε_j in each segment;
- 3. By consulting the stress-strain curves for the materials of the segments of the cross section, the values of the stresses σ_j and the moduli of elasticity E_i can be found;
- 4. From the new values of E_j and using Equation (3.1) a new location of the centroid can be determined and compared with the assumed position.

The above steps 2. through 4. may be repeated until satisfactory agreement is obtained bwtween assumed and calculated location of the centroid, and ε_j and σ_j will be accepted as strains and stresses in the section.

This iterative procedure is suggested by Zienkiewicz (11) as having been used by other investigators in dealing with nonlinear problems of structures. This process has been found to be convergent in most practical examples and generally three or four iterations of this type lead to an adequate solution.

Differential Equations of Motion

Having obtained the stresses σ_j at every segment of the section, thrusts, bending moments and shears can be determined by use of Equations (3.4), (3.5) and (3.8).

The equations for dynamic equilibrium of all joints can be established. From the freebody diagram of joint i, Figure 3.7(b), the following equilibrium conditions can be verified at time t:

$$\sum F_{xi} = 0$$

$$Q_{xi} + Q_{xdi}^{(t)} + T_{i} \cos \theta_{i} - T_{i-1} \cos \theta_{i-1} + V_{i} \sin \theta_{i} - V_{i-1} \sin \theta_{i-1} - m_{i} \ddot{u}_{di}^{(t)} = 0 ;$$

$$\sum F_{yi} = 0$$

$$Q_{yi} + Q_{ydi}^{(t)} + T_{i} \sin \theta_{i} - T_{i-1} \sin \theta_{i-1} + V_{i} \cos \theta_{i} + V_{i-1} \cos \theta_{i-1} - m_{i} \ddot{v}_{di}^{(t)} = 0 ;$$

where

 $Q_{xdi}^{(t)}$, $Q_{ydi}^{(t)}$ = magnitudes of horizontal and vertical applied dynamic forces at joint i at time t; $\ddot{u}_{di}^{(t)}$, $\ddot{v}_{di}^{(t)}$ = values of horizontal and vertical components of acceleration of mass m_i at time t.



(a) Freebody of Bar i



(b) Freebody of Joint i

Figure 3.7. Freebody Diagrams for Dynamic Solution

The above equations are the differential equations of motion and if all forces are known at time t, they can be solved for the accelerations:

$$\ddot{u}_{di}^{(t)} = \frac{1}{m_{i}} \left[Q_{xi} + Q_{xdi}^{(t)} + T_{i} \cos \theta_{i} - T_{i-1} \cos \theta_{i-1} \right]^{+} + V_{i} \sin \theta_{i} - V_{i-1} \sin \theta_{i-1} \right].$$
(3.18)

$$\ddot{v}_{di}^{(t)} = \frac{1}{m_i} \left[Q_{yi} + Q_{ydi}^{(t)} + T_i \sin\theta_i - T_{i-1} \sin\theta_{i-1} + V_i \cos\theta_i + V_{i-1} \cos\theta_{i-1} \right].$$
(3.19)

Numerical Integration of the Equations of Motion

At the beginning of the dynamic process, when a dynamic force is applied, it will transmit initial accelerations or initial velocities to the point masses of the structure. Initial dynamic displacements are null, because the structure is at rest, although deformed, under the static loads.

Subsequent values of dynamic displacements, velocities and accelerations are found by a step-by-step numerical integration of the differential equations of motion, using the "Beta Method" developed by Newmark (7).

If u_{di} and v_{di} are dynamic displacements of joint i in the X and Y directions, respectively, the numerical integration equations used in the Beta Method are the following, assuming a linear variation of accelerations with time, during a small time interval Δt :

$$\dot{u}_{di}^{(t+\Delta t)} = \dot{u}_{di}^{(t)} + \frac{\Delta t}{2} \ddot{u}_{di}^{(t)} + \frac{\Delta t}{2} \ddot{u}_{di}^{(t+\Delta t)};$$

$$\dot{v}_{di}^{(t+\Delta t)} = \dot{v}_{di}^{(t)} + \frac{\Delta t}{2} \ddot{v}_{di}^{(t)} + \frac{\Delta t}{2} \ddot{v}_{di}^{(t+\Delta t)};$$

$$u_{di}^{(t+\Delta t)} = u_{di}^{(t)} + \Delta t \dot{u}_{di}^{(t)} + \frac{(\Delta t)^2}{3} \ddot{u}_{di}^{(t)} + \frac{(\Delta t)^2}{6} \ddot{u}_{di}^{(t+\Delta t)} ;$$

$$v_{di}^{(t+\Delta t)} = v_{di}^{(t)} + \Delta t \dot{v}_{di}^{(t)} + \frac{(\Delta t)^2}{3} \ddot{v}_{di}^{(t)} + \frac{(\Delta t)^2}{6} \ddot{u}_{di}^{(t+\Delta t)} ;$$
(3.20)

where

and single dots mean the first derivatives of the displacements with respect to time, or velocities; and the double dots mean the second derivative of the displacements, or accelerations.

For the numerical integration procedure, it is assumed that the values of displacements, velocities and accelerations are known at time t, either as initial conditions of the problem or from the analysis of the preceding time interval Δt .

The accelerations of the masses at time t+ Δ t are assumed, and corresponding velocities and displacements are computed. A new configuration of the system is obtained and internal forces are calculated. By applying conditions of equilibrium at every joint, it is possible to compute the accelerations at time t+ Δ t. Calculated and assumed accelerations are compared and, if an acceptable agreement is obtained, the assumed accelerations are correct and the system is in its actual configuration; if not, an improved set of assumed accelerations equal to the newly calculated accelerations is used and the cycle is repeated, until the desired agreement is obtained. A new set of initial conditions is established and the computation proceeds for the next time interval.

The procedure may be outlined as follows:

- 1. The values of the accelerations are assumed at time t+ Δ t, equal to the accelerations at time t;
- Velocities and displacements at the end of time interval are calculated by Equation (3.20);
- The configuration of the structure at time t+At is determined by calculating new lengths and slopes of the bars, average strains and average curvatures;
- Strain and stress distributions are found for all selected cross sections, using the iterative process described previously, on page 25;
- Thrusts, bending moments and shears are calculated, using Equations (3.4), (3.5) and (3.8);
- Conditions of equilibrium of internal forces with static and dynamic loads are established at all joints, and resulting accelerations of the point masses are calculated, by applying Equations (3.18) and (3.19);
- 7. Steps 2. through 6. are repeated as many times as necessary to obtain a satisfactory agreement between assumed and calculated accelerations, but each time the assumed accelerations are taken equal to the previously calculated ones;
- 8. When a satisfactory agreement is obtained, the time is incremented by ∆t, the next time interval is determined, a new set of initial conditions is established, and the process is resumed, until the total response of the structure is obtained.

Influence of Time Interval

As it was pointed out by Newmark (7), the stability, convergence and accuracy of the numerical procedure depend not only on the characteristics of the structure, but mainly on the interval of time chosen for each set of calculations. With smaller intervals of time better solutions can be obtained, but a very short interval of time will lead to time consuming operations, even in a modern high speed computer. Also the larger the time interval the larger is the number of iterations required for convergence.

For the numerical integration equations here used, the mere convergence of the sequence of calculations is sufficient to insure stability and the rate of convergence will be an adequate criterion for the time interval, because if the time interval is chosen for convergence, the numerical procedure will always be stable.

Newmark also showed that the time interval must be related to the shortest period of vibration, or the period in the highest mode of vibration, for the lumped mass system, and the greater the number of masses the shorter will be the permissible time interval.

Convergence is assured if Δt is less than or equal to 0.389 T, where T is the smallest natural period of vibration. This expression may be used as a guide in the selection of the initial time interval, while not applicable to nonlinear problems, because as a structure goes into the inelastic range, in general the periods of vibration all become longer and the shortest period becomes longer as well. Consequently, the time interval can be considerably longer as plastic action develops in the structure. With continual re-examination of the time interval in terms

of the rate of convergence, the change in characteristics of the structure can be taken into account without loss in accuracy, and the results can be obtained with less computational work.

Sometimes difficulty arises in the determination of the smallest natural period of vibration, which may become a troublesome experience, almost impossible in more complex structures.

The best policy is to take a very small value for the initial time interval, and adjust this interval until an optimum value is obtained, thus maintaining the rate of convergence in four or five iterations for each interval of time. At the end of one cycle of iterations, if convergence is not reached in a prescribed number of iterations, the time interval is cut to a half of its value and everything starts again with the initial conditions at time t. If the operation is successfully completed, the time interval is multiplied by a factor, in order to adjust the time interval for the next set of calculations, making it smaller if the number of iterations required for convergence was greater than five, or larger if the required number of iterations was less than four, thus maintaining the rate of convergence in the optimum desired range of four or five iterations for each interval of time.

For circular arches, Veletsos et al. (9), Wolf (10) and Austin and Veletsos (1) used the classical frequency formula which leads to

$$T_k = \frac{2\pi}{C_k} S^2 \sqrt{\frac{\mu}{EI}}$$

where k represents the order of the mode of vibration and T_k is the corresponding period. The dimensionless constant C_k was shown to vary with the slenderness ratio of the arch, and was calculated for the ten lowest modes of vibration, and the highest value obtained was slightly

over 1,000 for very flexible fixed arches (slenderness ratio over 300), in the fifth antisymmetrical mode of vibration.

Using these data, a rough approximate value for the initial time interval can be obtained as

$$\Delta t = \frac{0.389 \times 2}{1000} S^2 \sqrt{\frac{\mu}{EI}} .$$

To make the interval of time even smaller to insure convergence from the very beginning of the numerical procedure, the following equation can be used

$$\Delta t = (2 \times 10^{-4}) \text{ s}^2 \sqrt{\frac{\mu}{\text{EI}}}$$

which assigns to the initial interval of time a value approximately twelve times smaller than that given by the preceding expression.

Dynamic Loads as Functions of Time

The dynamic forces $Q_{xdi}^{(t)}$ and $Q_{ydi}^{(t)}$ used in Equations (3.18) and (3.19) are variable with time and may be thought of as a product of a dimensionless function of time F(t) by a constant load parameter, i.e.:

$$Q_{xdi}^{(t)} = F(t) \times Q_{xdi}$$
;
 $Q_{ydi}^{(t)} = F(t) \times Q_{ydi}$.

The parameters Q_{xdi} and Q_{ydi} at any joint may result in a linear, a parabolic distribution, or a uniform pressure.

The function F(t) considered in this study may be represented by the triangular diagram in Figure 3.8(a), where F_m is the maximum value or peak, t_r is the time of rise, and t_d is the time of decay. The diagram of Figure 3.8(b) may be obtained from the first diagram by setting $t_r = 0$,



which will produce a force applied instantaneously at time t = 0 with its maximum value.

Other types of time functions may be easily formulated and taken into account, considering the particular problem to be solved.

Two types of dynamic loadings may be used: force pulses and impulses. For the first type of loading, the intensity of the force pulse at time t is given by

$$Q_{xdi}^{(t)} = F(t) \times Q_{xdi}$$
,
 $Q_{ydi}^{(t)} = F(t) \times Q_{ydi}$,

where F(t) is evaluated at each time, according to diagrams of Figure 3.8(a) or (b).

The initial conditions at time t = 0 for a force pulse are: Initial dynamic displacements and velocities:

$$u_{di}^{(0)} = 0 ; \qquad \dot{u}_{di}^{(0)} = 0 ; v_{di}^{(0)} = 0 ; \qquad \dot{v}_{di}^{(0)} = 0 .$$

Initial accelerations:

$$\ddot{u}_{di}^{(0)} = F(0) \times Q_{xdi}/m_i;$$

 $\ddot{v}_{di}^{(0)} = F(0) \times Q_{ydi}/m_i.$

For the second type of loading, F(t) is made equal to zero at all times, and consequently $Q_{xdi}^{(t)} = 0$ and $Q_{ydi}^{(t)} = 0$ at all times, and Q_{xdi} and Q_{ydi} have to have the nature of impulses.

The initial conditions at time t = 0 for an impulse are: Initial dynamic displacements and accelerations:

$$u_{di}^{(0)} = 0 ;$$
 $\ddot{u}_{di}^{(0)} = 0 ;$
 $v_{di}^{(0)} = 0 ;$ $\ddot{v}_{di}^{(0)} = 0 .$

Initial velocities:

$$\dot{u}_{di}^{(0)} = Q_{xdi}/m_i;$$

 $\dot{v}_{di}^{(0)} = Q_{ydi}/m_i.$

Adjustment of Static Results

For the static solution, as already stated, the bars are flexible and undergo small deformations, whereas for the dynamic solution, the bars are flexurally rigid, and since the dynamic displacements may be very large, the small displacement geometry is no longer applicable.

Also, the rotations of the transverse cross sections have to be concentrated at the joints, in order to make the bars flexurally rigid for the dynamic solution. When this operation is performed over the deformed structure, the equilibrium of the forces at the joints may well not be satisfied, resulting in unbalanced forces, generally small but large enough to interfere with the dynamic solution, by transmitting initial unwanted accelerations to the point masses, concentrated at the joints.

In order to eliminate the unbalanced forces at the joints, first a strain and stress distribution is obtained, by following the procedure described previously on page 25, using the average strains and curvatures caused by static loads and given by Equations (3.16) and (3.17).

This approach will also indicate, for further checking, whether the material at any segment went into the inelastic range during the static

process, or was kept in the linearly elastic behavior, with unchanged location of the centroid at all sections, as previously assumed for the static solution.

With the resulting stress distribution, thrusts, bending moments and shears can be determined with Equations (3.4), (3.5) and (3.8), and residual accelerations $\ddot{u}_{di}^{(r)}$ and $\ddot{v}_{di}^{(r)}$ can be obtained by applying Equations (3.18) and (3.19), where $Q_{xdi}^{(t)}$ and $Q_{ydi}^{(t)}$ are null.

The unbalanced forces at the joints can be calculated as

$$\Delta Q_{xi} = m_i \quad \ddot{u}_{di}^{(r)},$$

$$\Delta Q_{yi} = m_i \quad \ddot{v}_{di}^{(r)},$$

which are subtracted from the applied static loads.

It was verified that the disturbing forces ΔQ_{xi} and ΔQ_{yi} were very small, but have to be removed anyway, in order to leave all joints in equilibrium at the beginning of the dynamic process.

CHAPTER IV

COMPUTER PROGRAM

The method of analysis described in the preceding chapter was programmed in FORTRAN language for solution in the IBM/360 Model 65 computer of the Oklahoma State University. Only minor changes may become necessary to run the program in other types of computers having a FORTRAN compiler.

According to the program listing presented in Appendix B, a maximum of 24 bars is allowed for the replacement structure. By changing only the dimension statements, an increased number of bars may be taken into account, limited only by the size of the available computing system.

The object code requires approximately 100 K bytes and the array area is approximately 48 K bytes for a maximum of 24 bars, with an additional 5 K bytes required for every additional four bars. An array area slightly over 140 K bytes will be needed to store data and calculated results for a problem with a maximum of one hundred bars in the replacement structure. Double precision arithmetic is used for all real variables, with approximately 16 significant decimal digits.

The program, named DYNARCH, consists of a main driver program, 29 subroutines and one function subprogram. Although some subroutines could be included in the main program or easily incorporated in other subroutines, the program in subroutine form has more flexibility and

facilitates changes to take into consideration particular features of specific problems, if necessary.

A summary flow diagram of the program is presented in Figure 4.1. A guide for data input is presented in Appendix C.

The execution starts by reading the identification of the run followed by the identification of the problem. More than one problem can be solved in the same run. Execution terminates, if a card with the first four columns blank is inserted for the identification of problem. A termination message may be written on columns 5 to 80.

Problem data are read and echoed by subroutine INECHO. A function FX must be supplied by the user in order to describe the arch axis in the global coordinate system, starting at the origin. Any kind of continuous and single-valued function in the first quadrant may be used. Thus the program will handle circular, parabolic, catenary, sinusoidal, cycloidal axes, or others. Examples of function FX used in this study are presented in Appendices B and D.

An option for a broken line structure, such as portal frames or arches with initial out-of-roundness is offered. In this case, the coordinates of all joints of the model must be supplied and a dummy function FX must be used. Options for symmetry and for solving a problem only in the linearly elastic range are also offered.

Subroutine GEOM divides the structure described by FX into the specified number of bars with initially equal lengths and determines the coordinates of all joints, or, if the broken line structure option is used, reads the given coordinates of joints and calculated the lengths of all bars; then initial slopes and curvatures are also obtained and stored in common for further use.



Figure 4.1 Summary Flowchart of Program DYNARCH

Subroutine DIST, using other subroutines, distributes all input data and calculates concentrated masses, self-weight and applied static loads at the joints as the result of a linear, parabolic distribution or uniform pressure acting in some specified region of the structure. This subroutine, by calling subroutine CENTER, also determines the locations of centroids, axial and flexural stiffnesses at all 'joint sections. The area of the cross section is allowed to vary from joint to joint.

The static process is controlled by subroutine STATIC, for which the summary flow diagram is shown in Figure 4.2(a) and 4.2(b), on pages 42 and 43.

Subroutine SOLVE calculates the coefficients A_i , B_i , and C_i in equations (3.15), using the axial and flexural stiffnesses of the bars obtained in the local coordinate system by subroutine MSTIF. The independent coefficients D_i are set equal to the respective static forces with signs changed, as described in Chapter III. Then the system of linear simultaneous equations is solved for the displacements.

A refinement of the solution is obtained by an iterative procedure, where successive values of additional displacements are calculated, due to the residues of the equations in the system of simultaneous equations. The residues are computed by subroutine RESIDU and the procedure is considered to converge to the best possible solution for the displacements when the residues are kept smaller than a prescribed value of tolerance.

Subroutine STFOR is then called to perform the calculation of static forces at the ends of all bars and convert these forces into thrusts, bending moments and shears in the classical engineering sign convention. Also the reactions resulting from boundary conditions are calculated. Boundary conditions are taken into account by specified



ر و و در





displacements at any joint. Also hinges may be specified at any joint, as described in the guide for data input in Appendix C.

The problem is now ready to go into the dynamic process. Although the structure is assumed to behave linearly in the static process, a response of the structure in the inelastic range under static loads may be obtained by a small modification of the program, which would lead the structure into the dynamic process for just one step with zero dynamic load applied. This would cause a redefinition of location of centroids, axial and flexural stiffnesses, and the collapse of the structure under applied static loads would also be checked. With modified stiffnesses, the static process could be applied again, and a new configuration of the structure would result, with the corresponding internal forces. The operation could be repeated until a satisfactory agreement between two successive configurations of the structure were obtained. Since the response of the structure depends on the location of the centroid at all sections, the depths of the centroids could control the convergence of this iterative procedure.

As a matter of fact, there is no need to go into the dynamic process, but only into part of the preliminary step of adjustment of static results, as described previously, on page 36, and as soon as the locations of centroids, axial and flexural stiffnesses were redefined, and collapse checked, the iterative procedure described above could be applied, thus obtaining the inelastic response of the structure under static loads. Then, if desired, the dynamic loadings could be superimposed. This was not done here, since it was preliminarily assumed that the static process would be applied in the elastic range of behavior of the materials of the structure.

The dynamic process is controlled by subroutine DYNAM, for which the summary flow diagram is shown in Figure 4.3(a) through 4.3(d) on pages 46-49.

Since the dynamic process is allowed to be applied independently of the static process, a check is first made whether the static solution was required and, if the answer is positive, subroutine ADJUST performs the adjustments of the static results to conform the structure to the dynamic process. For convenience this subroutine also initializes the parameters of the stress-strain curves at all joints.

Time is set to zero and the initial time interval is calculated, if not supplied. Then dynamic displacements, velocities, accelerations are initialized at the beginning of time interval and the dynamic process follows that described in Chapter III. The flow diagrams on pages 46 through 49 are self-explanatory.

Determination of function F(t) at time t is made by subroutine FTIME, which is called if the dynamic response of the structure to a force pulse is desired. If the dynamic loading is an impulse, F(t) is made constantly null. The time functions used are described previously on page 33. This subroutine may be easily modified to allow other types of functions.

A new configuration of the structure is obtained by subroutine JCURVT, which calculates average curvatures at all joints, lengths, angle changes and average strains at all bars.

Subroutines FORCE and INTERN perform the iterative procedure of determining the new locations of the centroids of the transformed areas of the sections, redefine axial and flexural stiffnesses, and calculate



Process



Figure 4.3. (Continued)



Figure 4.3. (Continued)

<





Figure 4.3 (Continued)

bending moments at all joints, thrusts and shears at all bars, as shown by the flowchart on page 47.

Subroutine SEARCH is used by INTERN to examine the parameters of the stress-strain curves and determine the values for the moduli of elasticity and average stresses at the segments of the section.

Subroutine ACCEL checks the conditions of equilibrium at the joints and calculates the accelerations of all point masses.

After the dynamic solution has converged at the end of the time interval, the parameters of the stress-strain curves are redefined to account for possible permanent sets.

The collapse of the structure is then checked by subroutine FAIL, under the failure criteria of maximum horizontal displacement, maximum vertical displacement, maximum shear, interaction of thrusts and bending moments, and crushing of extreme fibers. If collapse occurred, the circumstances and locations of failure are printed. This subroutine can easily be modified to allow other failure criteria.

Output of final or partial results, either for the static process or for the dynamic process, is made through subroutine OUTPUT. Two printout options are possible: (a) the maximum values of thrusts, shears, bending moments, horizontal and vertical displacements, and where they occurred, and (b) a complete response with values of thrusts, shears in all bars and bending moments, horizontal and vertical displacements at the joints.

CHAPTER V

VERIFICATION OF PROGRAM

In order to illustrate the solution capability of the program and demonstrate its use, and also to verify the accuracy of the method of analysis, several problems have been solved, and the results compared with those obtained by conventional closed form solutions or by methods used by other investigators.

In this chapter, some of those problems are described and the solutions from the computer program are discussed. Sample coding listings for data input and selected printout sheets for all example problems are presented in Appendix D.

The following problems are discussed here (DTP stands for "DYNARCH Test Problems"):

- Example Problem DTP1: Static solution only.
 Static solution of a reinforced concrete circular arch, having an angle of opening of 180 degrees.
- 2. Example Problems DTP2: Dynamic solution only--elastic response. DTP2.1--Linearly elastic dynamic solution of a wide flange steel two-hinged circular arch, having an angle of opening of 87.21 degrees, subjected to a rectangular pressure pulse of infinite duration.

DTP2.2--Linearly elastic dynamic solution of a wide flange steel simply supported beam, subjected to a sinusoidal impulse loading.

3. Example Problems DTP3: Combined static and dynamic loading-inelastic response

> DTP3.1--Inelastic response of a prismatic reinforced concrete beam, under combined static and dynamic loadings DTP3.2--Inelastic response of the beam in the previous problem, with dynamic loading reversed.

Example Problem DTP1: Static Solution Only

For this problem, a two-hinged circular arch with an angle of opening of 180 degrees is considered. A central concentrated load P = -2000lb (downwards) is applied at the crown. The arch has a constant cross section of reinforced concrete with bottom and top reinforcement, as shown in Figure 5.1. The mechanical model consisted of 48 bars.

For a concentrated load at the crown, the closed form solution yields:

Horizontal reactions:

 $H = -P/\pi = 636.62$ lb.

Vertical reactions:

V = -P/2 = 1000.00 lb.

Moment at the crown:

 $M_c = (V - H)R = 6.4186 \times 10^4 \text{ in-lb.}$

Vertical displacement at the crown:

 $v_{c} = \frac{PR^{3}}{8EI} = \frac{3\pi^{2} - 8\pi - 4}{\pi} = -0.0418$ in.

These values are compared with the computer results in Figure 5.1. It is seen that a very good agreement is obtained.



Closed Form Solution	DYNARCH Solution
636.62	636.56
1000.00	1000.00
6.4186 x 10 ⁴	6.4196 x 10 ⁴
- 0.0418	- 0.0426
	Closed Form Solution 636.62 1000.00 6.4186 x 10 ⁴ - 0.0418

(c) Comparison of Results

Figure 5.1. Static Solution--Problem DTP1

Example Problems DTP2: Linearly

Elastic Response

Two problems were solved, where only linearly elastic properties of the material were considered: (1) a two-hinged circular arch and (2) a simply supported beam. For both problems no static loads were taken into account, then only the dynamic solution was required.

Problem DTP2.1--Two-Hinged Circular Arch

For this problem, a two-hinged circular arch with an angle of opening $\phi_0 = 87.21$ degrees is considered, Figure 5.2. No static loading is applied. The dynamic loading is taken in the form of a rectangular pressure pulse of infinite duration, uniformly distributed over the entire arch. A similar problem has been solved by Eppink and Veletsos (6).

The wide flange steel section W16X88 used here has a radius of gyration r = 6.78 inches. The span of the arch is $L_0 = 100r$ and the rise is $f = 0.2L_0$. The uniform pressure pulse has a magnitude $p_0 = 0.01$ p_{cr} , where p_{cr} is the critical buckling pressure of the arch corresponding to an antisymmetrical mode of deformation, defined as

$$p_{cr} = \begin{bmatrix} \frac{4\pi^2}{\phi_0^2} & -1 \end{bmatrix} = \frac{EI}{R^3}$$

The computer results are tabulated in Table D.1 in Appendix D for elapsed times t from 0.1 T_0 to 3.0 T_0 , where T_0 is the period of the "breathing" mode of vibration of a circular ring with the radius of the arch and is given by

$$T_o = 2\pi R \sqrt{\frac{\mu}{EA}}$$

à,



Figure 5.2. Data for Problem DTP2.1--Linearly Elastic Dynamic Solution Only

For this particular example $p_{cr} = 4711$ lb/in and T = 1.555 x 10^{-2} . sec.

By comparing the values in Table D.1 with those obtained by Eppink and Veletsos (6), one can see that the values for thrusts and vertical displacements at the crown compare very favorably with a difference between 0% and 2%. The values for bending moments at 1/4 point and at the crown present a higher distortion, generally less than 6%, probably due to a different approach in the calculation of bending moments.

Nevertheless, the values given for bending moments in Table D.1, on page 163, closely agree with those calculated by the formula for flexural vibration of a circular ring

$$M = \frac{EI}{R^2} \left[\frac{\partial^2 u_r}{\partial \theta^2} \right] + u_r$$

given by Timoshenko (8), where u_r is the radial displacement. This equation can be applied at the crown, considering a predominantly flexural mode of vibration of the arch near the midspan.

Taking the above equation and using the second order central finite difference equation as an approximation for the second derivative, the values of bending moments were calculated at the crown and also included in Table D.1 for comparison.

Problem DTP2.2--Simply Supported Beam

For this problem, a simply supported beam is considered, with the same wide flange steel section as the previous problem. No static load is applied. The dynamic loading is taken in the form of a sinusoidal impulse, as shown in Figure 5.3.



	Results Obtained by Dawkins (4)		Results From
Quantity	Closed Form	Program IMPBC	DYNARCH
Maximum moment at midspan (in-kip)	2.7803 x 10 ³	2.7675 x 10 ³	2.7543 x 10 ³
Maximum deflection at midspan (in)	- 1.7715	- 1.7885	- 1.7801
Natural period (sec)	0.1056	0.1062	0.1061

(b) Comparison of Results

Figure 5.3. Problem DTP2.2--Linearly Elastic Dynamic Solution Only

This problem was also solved by Dawkins (4), using both a closed form solution and his program IMPBC for analysis of beam-columns under impulse loadings.

4

By comparing the results summarized in Figure 5.3, one can see that there is a close agreement between the results obtained by program DYNARCH and those obtained by Dawkins. Differences are less than 1% from closed form solution and about 0.5% from program IMPBC.

> Example Problems DTP3: Inelastic Response

Two problems were also solved to illustrate the application of the computer program and method of analysis to problems where nonlinear properties of the materials are taken into account. The inelastic response of a reinforced concrete beam, under combined static and dynamic load-ings, was obtained (1) by superimposing a downward sinusoidal impulse to a uniform static load, and (2) by considering the same static load, but reversing the direction of the impulse.

Problem DTP3.1--Downward Sinusoidal Impulse

For this problem, a prismatic reinforced concrete beam on simple supports, shown in Figure 5.4, was subjected to a uniform static load and to an impulse loading varying sinusoidally over the length of the beam. Beam cross section, loadings and stress-strain curves for concrete and reinforcement are also shown in Figure 5.4. Ultimate strain for concrete is 0.003. Other failure parameters are listed in sample output sheets in Appendix D.


⊢ 8"

Figure 5.4. Data for Problem DTP3.1--Downward Sinusoidal Impulse

The solution indicated failure due to thrust-moment interaction near the center of the beam, at approximately 1.25 milliseconds after the impulse was applied.

Dawkins (4) solved a similar problem using program IMPBC. Numbers can not be compared, because (1) a smaller static load was used here, in order to keep the static solution within the elastic range of the materials, and (2) Dawkins did not consider the shifting of the centroid as inelastic action takes place. Nevertheless, the pattern of behavior of the beam in the two solutions was identical.

Problem DTP3.2--Upward Sinusoidal Impulse

The beam considered for this problem was the same for problem DTP3.1 with the direction of the dynamic loading reversed. This combination of static and dynamic loadings may be the result of a detonation in the interior of a building.

As cracks and consequently large strains should be expected at the top of the section, where no reinforcement was specified, the values in the positive side of the stress-strain curve for the concrete were exaggerated. Also very large values were taken for the maximum allowed negative bending moments in the interaction diagram.

The computer solution indicated failure by maximum prescribed vertical displacement, which occurred at the midspan, at approximately 4.5 milliseconds after the application of the impulse loading. At time of failure, the location of the centroid in the central portion of the span was indicated to be located below the center of gravity of the reinforcement.

CHAPTER VI

APPLICATION OF PROGRAM

In order to demonstrate the applicability of the method of analysis and of the program developed in this study, a two-hinged semi-circular arch of reinforced concrete, Figure 6.1(a), was analyzed under two conditions of dynamic loadings:

1. A forcing pressure pulse of sinusoidal variation, applied at the top of the arch, extending over a region of 30 degrees.

2. The same forcing pressure pulse, applied at the quarter point of the arch, over 30 degrees.

For both problems, the only static loading considered was the self weight of the arch.

Stress-strain diagrams for concrete and steel are shown in Figures 6.1(d) and (e). The time function F(t) was a triangular function with peak F_m , as shown in Figure 6.1(c), having time of rising t_r equal to half of total time of duration t_d .

The uniform cross section is shown in Figure 6.1(b). From stressstrain curves, initial tangent modulus for concrete is $E_c = 3.6 \times 10^6$ pounds per square inch, and for steel $E_s = 3.0 \times 10^7$ pounds per square inch. The transformed area of the cross section is 110.667 in² and the moment of inertia is 1386.667 in⁴, and the radius of gyration is r = 3.54 inches.







(b) Cross Section







The span was taken equal to $L_0 = 100 \text{ r} = 354 \text{ inches, which corresponds to a radius R = <math>L_0/2 = 177$ inches for the arch, and a slenderness ratio S/r = 157.08, or a value of R/r = 50 for the ratio of the radius of the arch to the radius of gyration of the cross section.

Problem DAP1: DYNARCH Application Problem 1

For this problem, the forcing pressure pulse was applied at the top of the arch, as shown in Figure 6.2(a).

At time t, the magnitude of the pulse is

 $p(t,\alpha) = p(\alpha) F(t) = P_0 \sin [6(\alpha + 15^\circ)] F(t)$

where α , given in degrees, measured from the vertical axis of symmetry, positive clockwise, varies from -15 to +15 degrees, as in Figure 6.2(a), and F(t) is the triangular time function shown in Figure 6.1(c).

The maximum value of the pressure pulse occurs at the crown of the arch at time t_r :

$$p_{max} = P_0 F_m$$

where ${\bf F}_{\rm m}$ is the peak of the time function.

The purpose of this analysis was to determine the minimum values of the pressure pulse, p_{mf} , capable of causing the failure of the arch for various times of duration.

A type of iterative procedure was used to attain this goal. For a given time of duration t_d , a forcing pulse of maximum value $p_{max} = p_0 F_m$ was initially applied. The program was run and if failure occurred, p_{max} was taken as an upper bound for p_{mf} , the minimum value capable of causing failure of the arch for that particular time of duration; if failure did not occur, p_{max} was taken as a lower bound for p_{mf} .



Figure 6.2. Dynamic Loadings for DYNARCH Application Problems

Subsequent values of p_{max} were tried until the lower and upper bounds were sufficiently close to each other, and the average of these bounds was taken as p_{mf} .

As an example, a time of duration $t_d = 1.0 \times 10^{-2}$ sec (10 milliseconds) was preliminarily considered and a load $p_{max} = 3000$ lb/in was applied. Failure occurred at time t = 6.3068 millisec, due to crushing of the top fibers of the section at the crown of the arch. A load $p_{max} =$ 2500 lb/in was then applied and failure occurred at time t = 8.18 millisec, and a new upper bound was established. A smaller load $p_{max} =$ 2000 lb/in was applied and failure did not occur. Then the last value was taken as a lower bound for p_{mf} . Having a lower and an upper bound for p_{mf} , the standard bisection method of analysis was followed until $p_{mf} =$ 2280 lb/in was considered an acceptable value for the minimum load causing failure and having a total time of duration of 10 milliseconds.

Other times of duration t_d for the forcing pressure pulse were considered and the results are summarized in the diagram of Figure 6.3, where the values of p_{mf} were plotted against t_d , curve AA.

For values of t_d smaller than 20 milliseconds, failure occurred after the pulse, and for greater values of t_d , failure occurred during the pulse, at times greater than 70% of the times of duration.

By applying a very small dynamic loading of the same nature, so that the response of the arch would be in the linearly elastic range, an approximate value for the period of vibration of the arch was obtained as 70 milliseconds, which was the maximum value considered for t_d .

For this problem, a model with 24 bars was considered, but advantage was taken of symmetry of shape and loading, and only half the structure was solved. Failure always occurred by crushing of the top fibers of





the section at the crown, with ultimate strain prescribed as 0.003. Samples of data input and output are shown in Appendix D.

The maximum time spend in the computer for one loading was 16.7 minutes for $t_d = 7.0 \times 10^{-2}$ sec (70 millisec), with time limit of 7.2 x 10^{-2} sec, the solution progressing at an average of 72 microseconds of elapsed time per second of computer time. For smaller times of duration this average sometimes was as low as 50 microseconds per second.

Problem DAP2: DYNARCH

Application Problem 2

For this problem, the forcing pressure pulse was applied at the quarter region of the arch, as shown in Figure 6.2(b).

At time t, the magnitude of the pulse is

 $p(t,\alpha) = p(\alpha) F(t) = p_0 \sin [6(\alpha + 15^\circ)] F(t)$

where α varies from -15 to +15 degrees and is measured from the radius at the quarter point; F(t) is the triangular time function shown in Figure 6.1(c).

The maximum value of the pressure occurs at the quarter point of the arch at time t_r .

The same procedure applied to Problem DAP1 was used to find the variation with time of duration of p_{mf} , the minimum value of the pressure to cause failure of the arch, and the results are also symmarized in the diagram of Figure 6.3, where the values of p_{mf} were plotted against t_d , curve BB.

By comparing the results with those obtained for the load applied at the top, one can see that, except for smaller values of t_d , when the loadings have more localized effects at the region of their application, a higher load has to be applied at the quarter region of the arch in order to cause failure, which should be expected.

It was verified that failure occurred during the pulse, at times varying from 60% to 90% of times of duration, this percentage being smaller for larger times of duration.

A model with 24 bars was considered, but in this cause advantage could not be taken of the symmetry of the structure, because the load was not symmetrical. Failure always occurred by crushing of the top fibers of the section at the quarter point, with prescribed ultimate strain of 0.003.

The computer solutions progressed at rates between 30 and 40 microseconds of elapsed time per second of computer time, this rate being smaller for smaller times of duration.

CHAPTER VII

SUMMARY, CONCLUSIONS AND RECOMMENDATIONS

Summary

A method of analysis of arches, beams and portal frames, under dynamic loads or combined static and dynamic loads has been developed in this study, by using a discrete-element mechanical model, representing the actual structure.

For the static loadings, when considered, the deformations of the structure are assumed to be sufficiently small so that conventional matrix analysis of structures is employed. Dynamic effects are superimposed on static results.

The method developed herein takes into account material nonlinearity combined with geometric nonlinearity due to large displacements, by constantly revising the axial and flexural stiffnesses at selected sections of the structure, as deformations occur under transient loads. This is accomplished by considering the modification of the transformed area of the general cross section and consequent relocation of its centroid, as plastification and/or cracks take place. A variable modulus of elasticity, the secant modulus, is used. Failure criteria are also established.

The integration of the equations of motion is made by using a stepby-step iterative numerical procedure, the Newmark's Beta Method, based

on the assumption of linear variation of the acceleration during the time-step interval.

A new configuration of the replacement structure is obtained and internal forces are calculated at the end of every time-step interval.

Convergence and stability of the dynamic process are insured by redefining the time-step interval after every cycle of iterations.

Dynamic loadings may be taken in the forms of impulses or time dependent pulses.

A computer program was written in FORTRAN language for solution in the IBM/360 Model 65 computer of the Oklahoma State University. The program in subroutine form is sufficiently general to handle a large variety of parameters, such as material properties, structure shapes and cross sections, collapse criteria, and can easily be changed to take into consideration particular features of specific problems.

In order to illustrate the solution capability of the program and verify the accuracy of the method of analysis, several problems have been solved, and the results compared with known solutions.

The program was used in two application problems to determine a minimum sinusoidal pressure pulse with varying time of duration, capable of inducing collapse of a two-hinged semi-circular arch, when applied over a limited region of the arch.

Conclusions /

From the present study the following conclusions can be drawn:

 The computer results for linearly elastic problems compared satisfactorily with those obtained by conventional closed form solutions, or by methods used by other investigators.

2. The continuous relocation of the centroid of the transformed area of the cross section constitutes a realistic form of treating nonlinear problems of structures, and provides a very useful means of redefining axial and flexural stiffnesses as inelastic action takes place.

3. The application problems showed that the program developed can be advantageously used to perform parametric studies in dynamic problems of arches, beams and portal frames, with geometric and material nonlinear characteristics, to determine position, magnitude and time of duration of dynamic loadings capable of inducing collapse of a given structure.

Recommendations

Future extensions of the model and program may include the effects of rotatory inertia and shear deformation, as well as prestressing of concrete members.

Experimental research could be performed, which would have the purpose of evaluating the analytical results obtained with the program. Carefully controlled laboratory or field tests, even in a limited number, would be very helpful in the evaluation of the method of analysis, and would have the merit of supplying additional information on material behavior under dynamic loadings.

A SELECTED BIBLIOGRAPHY

- (1) Austin, Walter J., and A. S. Veletsos. "Free Vibration of Arches Flexible in Shear." <u>Journal of the Engineering Mechanics</u> <u>Division</u>, ASCE, Vol. 99 (August, 1973), pp. 735-753.
- (2) Blaauwendraad, J. "A Realistic Analysis of Reinforced Concrete Framed Structures." <u>Heron</u>, Vol. 18, No. 4 (1972). Dept. of Civil Engineering, Technological University of Delft, The Netherlands.
- (3) Dawkins, William P. "Analysis of Tunnel Liner-Packing Systems." <u>Journal of the Engineering Mechanics Division</u>, ASCE, Vol. 95, No. EM3 (June, 1969), pp. 679-693
- (4) Dawkins, William P. "A Method of Analysis for Reinforced Concrete Beam-Columns Subjected to Impulse Loading." Report Submitted to Oklahoma State University (Preliminary Copy), March, 1971.
- (5) Den Hartog, J. P. "The Lowest Natural Frequency of Circular Arcs." <u>Philosophical Magazine</u>, Series 7, Vol. 5, No. 28 (February, 1928), pp. 400-408.
- (6) Eppink, R. T., and A. S. Veletsos. "Dynamic Analysis of Circular Elastic Arches." <u>Proceedings</u> of the Second Conference on Electronic Computation, ASCE, Pittsburg, September, 1960, pp. 477-502.
- (7) Newmark, N. M. "A Method of Computation for Structural Dynamics." Transactions, ASCE, Vol. 127, Part I (1962), pp. 1406-1433.
- (8) Timoshenko, S. <u>Vibration Problems in Engineering</u>. Princeton, New Jersey: D. Van Hostrand Co., Inc., 1955.
- (9) Veletsos, A. S., et al. "Free In-Plane Vibration of Circular Arches." <u>Journal of the Engineering Mechanics Division</u>, Proceedings, ASCE, Vol. 98, No. EM2 (April, 1972), pp. 311-329.
- (10) Wolf, Joseph A. "Natural Frequencies of Circular Arches." <u>Journal</u> <u>of the Structural Division</u>, Proceedings, ASCE, Vol. 97, No. ST9 (September, 1971), pp. 2337-2348.
- (11) Zienkiewicz, O. C. <u>The Finite Element Method in Structural and</u> <u>Continuum Mechanics</u>. New York: McGraw-Hill Publishing Co., 1967.

APPENDIX A

SOLUTION OF SIMULTANEOUS EQUATIONS

The system of linear simultaneous equations, on page 21, obtained for the static solution, is solved by a variation of Gauss elimination known as the recursion-inversion procedure.

First it will be shown that the general Equation (3.14) of the system Equation (3.15), on page 21, can be written in the form:

$$U_{i} = \alpha_{i+1} + \beta_{i+1} U_{i+1} .$$
 (A.1)

If Equation (A.1) is supposed to be valid, the following expression for U_{i-1} is also valid:

$$U_{i-1} = \alpha_i + \beta_i U_i . \tag{A.2}$$

Taking Equation (A.2) into (3.14), it results:

$$A_{i} (\alpha_{i} + \beta_{i}U_{i}) + B_{i} U_{i} + C_{i} U_{i+1} + D_{i} = 0$$
.

Solving for U_i:

$$U_{i} = -(A_{i\beta_{i}} + B_{i})^{-1} (A_{i\alpha_{i}} + D_{i}) - (A_{i\beta_{i}} + B_{i})^{-1} C_{i} U_{i+1}$$

Then the coefficients α_{i+1} and β_{i+1} of Equation (A.1) can be identified as:

$$\alpha_{i+1} = -(A_{i}\beta_{i} + B_{i})^{-1} (A_{i}\alpha_{i} + D_{i}); \qquad (A.3)$$

$$\beta_{i+1} = -(A_i\beta_i + B_i)^{-1} C_1 . \qquad (A.4)$$

Since A_i , B_i , C_i , D_i are known, the values of α and β at one station depend only on the values of α and β at previous station.

The first equation of system Equation (3.15) is:

 $B_1 U_1 + C_1 U_2 + D_1 = 0$

or

$$U_{1} = -B_{1}^{-1} D_{1} - B_{1}^{-1} C_{1} U_{2}$$

where

$$B_{1} = S_{1,1}^{1}, \quad C_{1} = S_{1,2}^{1} \quad \text{and} \quad D_{1} = -Q_{1}.$$

From Equation (A.1):

$$U_1 = \alpha_2 + \beta_2 U_2$$
.

Then

$$\alpha_2 = -B_1^{-1}D_1$$
 and $\beta_2 = -B_1^{-1}C_1$

For purposes of computer programming, it is useful to notice that the last two values can be obtained from Equations (A.3) and (A.4), by setting $\alpha_1 = 0$ and $\beta_1 = 0$, or, which is the same, by setting $A_1 = 0$, as it becomes evident from the first equation of system Equation (3.15).

Subsequent values of α and β can be calculated, until station m = n+1 is reached and U_m is established:

$$U_{m} = \alpha_{m+1} + \beta_{m+1} U_{m+1} .$$
 (A.5)

The last equation of system Equation (3.15) is

 $A_m U_n + B_m U_m + D_m = 0$

from which it is seen that $C_m = 0$ and Equation (A.4) produces $\beta_{m+1} = 0$. Then, from Equation (A.5):

$$U_m = \alpha_{m+1}$$

and all values of U_{i-1} can be found from Equation (A.2), starting with i = m and proceeding backwards until i = 2, when the last vector of displacements U_1 can be determined.

In order to improve the accuracy of the static solution, the values of the displacements are substituted into Equation (3.15), which must be satisfied. Generally, although the program deals with real variables in double precision, the first set of displacements do not satisfy entirely the system of Equation (3.15), which yields some residues R_i in each equation, i.e.:

$$A_i U_{i-1}^{(1)} + B_i U_{i-1}^{(1)} + C_i U_{i+1}^{(1)} + D_i = R_i^{(1)} \neq 0$$
,

the superscript (1) meaning that the values refer to the first trial solution obtained.

Then the entire procedure is repeated using $R_i^{(1)}$ in place of D_i , resulting in the general equation:

$$A_i U_{i-1}^{(2)} + B_i U_i^{(2)} + C_i U_{i+1}^{(2)} + R_i^{(1)} = 0$$
.

The new system is solved and a new set of displacements $U_i^{(2)}$ results in this second operation.

By adding the last two equations together, the residues are eliminated, resulting in:

$$A_{i} (U_{i-1}^{(1)} + U_{i-1}^{(2)}) + B_{i} (U_{i}^{(1)} + U_{i}^{(2)}) + C_{i} (U_{i+1}^{(1)} + U_{i+1}^{(2)}) + D_{i} = 0.$$

It is seen that $U_i = U_i^{(1)} + U_i^{(2)}$ is a better solution. The process goes on until all residues are kept equal or below a specified limit, as close to zero as possible.

Taking the displacements into Equation (3.11), the values of the forces at the ends of all bars can be determined. These forces result in matrix analysis sign convention, Figure 3.5(d), and to calculate bend-ing moments, shears and thrusts in classical engineering sign convention, Figure 3.5(f), the following transformation equations are used:

$$M_{i,i+1} = -f_{zi}^{i};$$

 $M_{i+1,i} = f_{z(i+1)}^{i};$

$$T_{i} = -f_{xi}^{i} \cos \theta_{i} - f_{yi}^{i} \sin \theta_{i};$$

$$V_{i} = f_{yi}^{i} \cos \theta_{i} - f_{xi}^{i} \sin \theta_{i}$$

or, in matrix form:

$$\begin{cases} T_{\text{bar i}} \\ V_{\text{bar i}} \\ M_{\text{joint i}} \end{cases} = - \begin{bmatrix} \cos_{i} & \sin_{i} & 0 \\ \sin_{i} & -\cos_{i} & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad \begin{cases} f_{xi}^{1} \\ f_{yi}^{i} \\ f_{zi}^{i} \end{cases}$$

Values of thrusts and bending moments will be taken into Equations (3.16) and (3.17), in order to calculate the average static strain in bar i and the average static curvature at joint i.

APPENDIX B

ν.

LISTING OF PROGRAM DYNARCH

С MAIN PROGRAM DYNARCH С С * * * * * * * * * * * С PROGRAM DYNARCH С С С LANGUAGE: FORTRAN IV С С COMPUTER: IBM /360 MODEL 65 С * JERSON DUARTE GUIMARAES С * PROGRAMMER: С PROFESSOR OF CIVIL ENGINEERING UNIVERSIDADE FEDERAL DE GOIAS С С GOIANIA, GOIAS, BRAZIL С * ANALYSIS AND PREDICTION OF COLLAPSE С **PURPOSE:** OF ARCHES, BEAMS AND PORTAL FRAMES, С \$ UNDER DYNAMIC LOADS OR COMBINED С * С STATIC AND DYNAMIC LOADINGS С С DETAILED INFORMATION CAN BE FOUND IN: * ****A METHOD OF ANALYSIS FOR NONLINEAR** С * DYNAMIC RESPONSE OF ARCHES!! C С PH.D. DISSERTATION BY JERSON DUARTE GUIMARAES С С OKLAHOMA STATE UNIVERSITY С JULY, 1974 С * * * * * * * * * * * С С IMPLICIT REAL * 8(A-H, O-Z) COMMON /IDENT/ ID1(40), ID2(19), NPROB COMMON /TABL1/ GRAV,TLIM,<EEP(7),ISTAT,ISOPT,NDL,IDOPT,NOUT,ISELFW DATA IBLANK, IYES / 4H , 3HYES / C 1000 FORMAT (20A4) 2000 FORMAT (//24X, 19A4) C READ RUN IDENTIFICATION - TWO CARDS C>--> READ 1000, ID1 C>--> READ PROBLEM IDENTIFICATION - ONE CARD 100 READ 1000, NPROB, ID2 С C>--> CHECK PROBLEM NAME AND STOP IF BLANK IF (NPROB .NE. IBLANK) GO TO 110 (>->)PRINT TERMINATION MESSAGE AND STOP PRINT 2000, ID2 STOP С CALL SUBROUTINE INECHO TO READ IN AND ECHO PROBLEM DATA C>--> 110 CALL INECHO CALL SUBROUTINE DIST TO GENERATE AND DISTRIBUTE DATA C>--> CALL DIST С CHECK WHETHER STATIC SOLUTION IS REQUIRED C>--> IF (ISTAT .NE. IYES) GD TO 120 CALL SUBROUTINE STATIC TO SOLVE FOR STATIC DISPLACEMENTS C>--> AND INTERNAL FORCES AND PRINT STATIC RESULTS С CALL STATIC C C>--> CHECK WHETHER DYNAMIC SOLUTION IS REQUIRED 120 IF (NDL .EQ. 0) GO TO 100 CALL SUBROUTINE DYNAM TO SOLVE FOR DYNAMIC DISPLACEMENTS C>--> INTERNAL FORCES, VELICITIES AND ACCELERATIONS, CHECK FOR С FAILURE AND PRINT DYNAMIC RESULTS С CALL DYNAM С C>--> RETURN FOR A NEW PROBLEM GO TO 100 END

C SUBROUTINE INECHO С С С С THIS SUBROUTINE READS AND ECHOES INPUT DATA FOR PROGRAM DYNARCH. IT ALSO CALLS SUBROUTINE SEDM TO SET UP INITIAL GEOMETRY OF THE STRUCTURE С С * С * С DATA ARE ORGANIZED IN TABLES, AS FOLLOWS: C * TABLE 1 С CONTROL DATA CROSS SECTION DESCRIPTION -C TABLE 2 ¢, С STRESS-STRAIN CURVES * TABLE 3 -TABLE 4 SPECIFIED CONDITIONS C С ź TABLE 5 -STATIC LOADS C * TABLE 6 -DYNAMIC LOADS COLLAPSE PARAMETERS TABLE 7 C -¢, C TABLE 8 -STATION COORDINATES Ĉ С * * * * * * * * * * * * C SUBROUTINE INECHO С IMPLICIT REAL # 8(A-H, O-Z) EXTERNAL FX COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE COMMON /CURVS/ EPSN(10,5),SIGN(10,5),EPSMUL(5),SIGMUL(5),EPSPR(5) COMMON /FAIL1/ SMAX(25), SMAXN(10), JS7N(10), UMAX, VMAX, NST7 COMMON /FAIL2/ BMUL(25), PMUL(25), BMULN(10), PMULN(10), PIAN(9), BIAN(9), EPSU(5), JIA7(10), NIA7 1 COMMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9) COMMON /IGEOM/ PHID(25), THD(24), IBRK, ISYM COMMON /TABL1/ GRAV, TLIM, KEEP(7), ISTAT, ISOPT, NDL, I) OPT, NOUT, ISELFW COMMON /TABL2/ DN(9,10), B1N(10),B2N(13),B3N(10), DTN(10),ATN(10), DBN(10), ABN(10), JSN(10), JRN(10), NCT2, NRT2 COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26), 1 JSDN(10), KU(10), KV(10), KTH(10), NCT4 1 COMMON /TABL5/ JI5(20), QXI5(20), QYI5(20), JM5(20), QXM5(20), 1 QYM5(20), JL5(20), QXL5(20), QYL5(20), NCT5 COMMON /TABL 6/ JI6(20), QXI6(20), QYI6(20), JM6(20), QXM6(20), QYM6(20), JL5(20), QXL6(20), QYL6(20), NCS(20), NPDL COMMON /TIMEF/ FO, TR, TD, FT, TIME, DTIME, TDTIME, IND, INTVL DIMENSION II(7) 1010 FORMAT (5X, 6(1X, A4), 2X, A3, 615/5X, 415, 5X, 4E10.3) 1020 FJRMAT (5X, 15, 20X, 3E10.3, 2X, A3) 1030 FORMAT (10X, 3(215, E10.3)) 1040 FORMAT (5X, 15, 10X, 4E10.3) 1050 FORMAT (20X, 3E10.3) 1060 FORMAT (5X, 15, 7X, 311, 10X, 3E10.3, 2X, A3) 1070 FORMAT (10F8.0) 1080 FORMAT (3(15, 2E10.3), 2X, A3) 2020 FORMAT (//20X, 23HTABLE 1. - CONTROL DATA//) 2030 FORMAT (31X, 25HNO KEEP OPTIONS EXERCIZED) 2040 FORMAT (31X, 25HRETAIN PRIOR DATA TABLES , 11, 5(2H, , 11)) 2050 FORMAT (/31X, 27HSTATIC SOLJTION REQUIRED: , A3//31X, 25HACCELERATION OF GRAVITY , 1PD10.3) 1 2051 FORMAT (/31x, 45HAXIS OF STRUCTURE DESCRIBED BY A SMOOTH CURVE) 2052 FORMAT (/31X, 21HBROKEN LINE STRUCTURE) 2054 FORMAT (/31X, 24HSONSYMMETRICAL STRUCTURE) 2055 FORMAT (/31X, 45HSYMMETRICAL STRUCTURE, NONSYMMETRICAL LOADING) 2058 FORMAT (/31X, 33HSYMMETRICAL STRUCTURE AND LOADING//31X, 1 27HSOLUTION FOR HALF STRUCTURE) 2060 FORMAT (/31X,21HSTATIC DUTPJT OPTION , 13X, 11) 2062 FORMAT (/31X, 24HSELF WEIGHT NOT INCLUDED) 2065 FORMAT (/31X, 33HSELF WEIGHT ADDED TO STATIC LOADS) 2070 FORMAT (/31X,26HNUMBER OF DYNAMIC LOADINGS, 6X, I3//31X, 1 21HDYNAMIC OUTPUT OPTION, 13X,11//31X,15HOUTPUT INTERVAL, 18X, 12//31X, 10HTIME LIMIT, 15X, 1PD10.3) 2 2072 FORMAT (/31X, 38HTYPE OF DYNAMIC LOADING: FORCING PULSE)

SUBROUTINE INECHO (CONTINUED) C 2074 FORMAT (/31X, 32HTYPE OF DYNAMIC LOADING: IMPULSE) 2076 FORMAT (/31X, 43HSOLUTION FOR LINEARLY ELASTIC RESPONSE ONLY) 2078 FORMAT (/31X, 27HINELASTIC RESPONSE REQUIRED) 2080 FORMAT (/31X,13HTIME INTERVAL, 12X, 1PD10.3) 2090 FORMAT (/31X, 35HTIME INTERVAL INTERNALLY CALCULATED) 2100 FORMAT (//20X,36HTABLE 2. - CROSS SECTION DESCRIPTION) 2110 FORMAT (/30X, 32HUSING DATA FROM PREVIOUS PROBLEM/) 2120 FORMAT (//30X, 12HCONTROL DATA//40X, 4HSTA., 10X, 36HTOP FLANGE WEB BOT FLANGE/41X, 3HNO., 13X, 31HWIDTH THICKNESS 1 WIDTH//40X, I3, 8X, 1P 3D13.4) 2 2130 FORMAT (//30X, 29HSEGMENT, MATERIAL, DEPTH DATA//37X, 58HSEG MAT DEPTH SEG MAT DEPTH SEG MAT DEPTH //(36X, 3(2I4, 1PD12.3))) 2140 FORMAT (//30X, 25HREINFORCEMENT DESCRIPTION//40X, 4HSTA., 6X, 17HTOP REINFORCEMENT, 5X, 20HBOTTOM REINFORCEMENT/41X, 1 3HNO., 7X, 14HDEPTH AREA, 10X, 14HDEPTH AREA/) 2150 FJRMAT (40X, I3, 1X, 2(4X, 1P2D10.3)) 2160 FORMAT (//20X, 31HTABLE 3. - STRESS-STRAIN CURVES//) 2170 FORMAT (27X, 10HCURVE NO. ,11//33X, 25HMATERIAL SPECIFIC WEIGHT 1PD12.3/33X, 28HCOMPRESSIVE ULTIMATE STRAIN , 1PD10.3/33X, 1 28HSTRESS VALUE SCALE FACTOR 2 , 1PD10.3/33X, 3 28HSTRAIN VALUE SCALE FACTOR , 1PD10.3/) 2180 FJRMAT (30X, 19HSTRESS INPUT VALUES/30X, 10F7.3/) , 1PD10.3/1 2190 FORMAT (30X, 19HSTRAIN INPUT VALUES/30X, 10F7.3/) 2200 FJRMAT (//20X,31HTABLE 4. - SPECIFIED CONDITIONS//) 2210 FORMAT (30X, 10HSTA. CJDE, 11X, 1 23HSPECIFIED DISPLACEMENT/49X, 1HU, 11X, 1HV, 10X, 3HR0T/) 2220 FJRMAT (30X, I3, 4X, 3I1, 28X, 1PD12.3) 2230 FORMAT (30X, 13, 4X, 311, 16X, 1P2D12.3) 2240 FORMAT (30X, 13, 4X, 311, 4X, 1P3D12.3) 2250 FORMAT (30X, 13, 4X, 311, 4X, 1PD12.3, 12X, 1PD12.3) 2260 FORMAT (30X, 13, 4X, 5HHINGE) 2280 FORMAT (//40X, 27HPARAMETERS OF TIME FUNCTION//45X, 1 21HTIME OF RISING: TR =, 1PD12.4/45X, TD =, 1PD12.4/45X, FO =, 1PD12.4) 21HTIME OF DECAY: 2 21HMAXIMUM VALUE: 3 2290 FORMAT (///20X, 23HTABLE 5. - STATIC LOADS/) 2300 FORMAT (/35X, 7HINITIAL, 14X, 12HINTERMEDIATE, 17X, 5HFINAL/26X, 4HSTA., 5X, 1HX, 9X, 1HY, 5X, 4HSTA., 5X, 1HX, 9X, 1HY, 5X, 4HSTA., 5X, 1HX, 9X, 1HY/) 1 2 2310 FORMAT (25X, 14,1X,1P2D10.3, 14,1X,1P2D10.3, 14,1X,1P2D10.3) 2320 FORMAT (/35X, 32HADDITIONAL DATA FOR THIS PROBLEM/) 2340 FORMAT (/35X, 4HNONE/) 2330 FORMAT (/35X, 22HUNIDENTIFIABLE LOADING//35X, 20HPROGRAM INTERRUPTED.) 1 2350 FORMAT (///20X, 26HTABLE 6. - DYNAMIC LOADING/) 2360 FORMAT (/31X, 20HDYNAMIC LOADING NO. , 12/) 2370 FORMAT (//20X, 30HTABLE 7. - COLLAPSE PARAMETERS/) 2380 FORMAT (/31X, 19HDISPLACEMENT LIMITS//40X, 25HMAXIMUM HORIZONTAL DISPL:, 1PD13.4/40X, 1 25HMAXIMUM VERTICAL DISPL:, 1PD13.4) 2 2390 FJRMAT (/31X, 12HSHEAR LIMITS/40X, 4HTERM, 13X, 5HSHEAR/40X, 1 4HSTA., 13X, 5HVALUE) 2400 FORMAT (/40X, I3, 7X, 1P2D15.4) 2410 FORMAT (/31X, 24HINTERACTION DIAGRAM DATA//40X 4HTERM,10X, MOMENT/40X, 4HSTA., 11X, 1 24HAXIAL FORCE 25HMULTI PLIER MULTIPLIER) 2 2420 FORMAT (/37X, 24HAXIAL FORCE INPUT VALUES/37X, 9F7.3) 2430 FORMAT (/37X, 19HMDMENT INPUT VALUES/37X, 9F7.3) 2440 FORMAT (//20X, 30HTABLE 8. - STATION COORDINATES//40X, 4HSTA., 12X, 7HX-COORD, 8X, 7HY-COORD) С INITIALIZE CONSTANT VALUES C>--> DATA IEND, KEEPI, IYES / 3HEND, 4HKEEP, 3HYES / ZER0 = 0.0000P5 = 5.00 - 01ONE = 1.0D00

С SUBROUTINE INECHO (CONTINUED) C ESTABLISH THE HEADING ON THE FIRST PAGE OF OUTPUT C>--> CALL HEADNG READ AND ECHO TABLE 1 - PROBLEM CONTROL DATA C>--> READ 1010, (KEEP(I), I=2,7), ISTAT, ISELFW, ISOPT, NADL, LDTYPE, IDOPT, NJUT, INEL, IBRK, ISYM, NB, SPAN, GRAV, TLIM, DTIME 1 NJ = NB + 1C>--> CALL SUBROUTINE GEOM TO CALCULATE OR READ STATION COORDINATES CALL GEOM (FX) J = 0K = 1 DO 110 I = 2, 7 II(K) = 0IF (KEEP(I) .NE. KEEPI) 30 TO 110 II(K) = IJ = J + 1K = K + 1110 CONTINUE PRINT DATA IN TABLE 1 C>--> **PRINT 2020** IF (J .GT. 0) GO TO 120 PRINT 2030 GD TO 130 120 PRINT 2040, (II(I), I = 1, J) 130 PRINT 2050, ISTAT, GRAV IF (IBRK .EQ. 0) PRINT 2051 IF (IBRK .EQ. 1) PRINT 2052 IF (ISYM .EQ. 0) PRINT 2054 IF (ISYM .EQ. 1) PRINT 2056 IF (ISYM .EQ. 2) PRINT 2058 IF (ISTAT .NE. IYES) GO TO 140 PRINT 2060, ISOPT IF (ISELFW .EQ. 0) PRINT 2062 IF (ISELFW .EQ. 1) PRINT 2065 140 IF (KEEP(6) .EQ. KEEPI) GJ TJ 145 NDL = 0145 NDL = NDL + NADLIF (NDL .EQ. 0) GO TO 160 PRINT 2070, NDL, IDOPT, NOUT, TLIM IF (LDTYPE .EQ. 0) PRINT 2072 IF (LDTYPE .EQ. 1) PRINT 2074 IF (INEL .EQ. 0) PRINT 2076 IF (INEL .EQ. 1) PRINT 2078 IF (DTIME .EQ. ZERO) GO TO 150 PRINT 2C80, DTIME GO TO 160 150 PRINT 2090 С C>--> READ AND ECHO TABLE 2. - CROSS SECTION DESCRIPTION C 160 CALL HEADNG **PRINT 2100** IF (KEEP(2) .EQ. KEEPI) GO TO 210 J = 1170 READ 1020, JSN(J), B1N(J), B2N(J), B3N(J), IENDN DD 180 I = 1, 9, 3 180 READ 1030, IS(I), MAT(I), DN(I,J), IS(I+1), MAT(I+1), 1 DN(I+1, J), IS(I+2), MAT(I+2), DN(I+2, J) IF (IENDN .EQ. IEND) GO TO 190 J = J + 1GO TO 170 NCT2 = J190 NRT2 = 1200 READ 1040, JRN(NRT2), DTN(NRT2), ATN(NRT2), DBN(NRT2), ABN(NRT2) IF (JRN(NRT2) .EQ. JSN(NCT2)) GO TO 220 NRT2 = NRT2 + 1GO TO 200 210 PRINT 2110

```
C
     SUBROUTINE INECHO
                                                                (CONTINUED)
C.
  220 DO 230 I = 1, NCT2
         PRINT 2120, JSN(I), B1N(I), B2N(I), B3N(I)
         PRINT 2130, (IS(J), MAT(J), DN(J,I), J = 1, 9)
  230 CONTINUE
      PRINT 2140
      PRINT 2150, (JRN(I), DTN(I), ATN(I), DBN(I), ABN(I), I = 1, NRT2)
C
C>-->
         READ AND ECHO TABLE 3. - STRESS-STRAIN CURVES
C
      CALL HEADNG
      PRINT 2160
      IF (KEEP(3) .EQ. KEEPI) GJ TJ 250
      READ 1020, NSSC
      DJ 240 I = 1, NSSC
         READ 1050, SIGMUL(I), GAMMA(I)
         READ 1070, (SIGN(J,I), J = 1, 10)
         READ 1050, EPSMUL(I), EPSU(I), EPSPR(I)
         READ 1070, (EPSN(J,I), J = 1, 10)
         EPSU(I) = DABS(EPSU(I))
  240 CONTINUE
  GO TO 260
250 PRINT 2110
  260 D0 270 I = 1, NSSC
         PRINT 2170, I, GAMMA(I), EPSU(I), SIGMUL(I), EPSMUL(I)
         PRINT 2180, (SIGN(J,I), J = 1, 10)
         PRINT 2190, (EPSN(J, I), J = 1, 10)
  270 CONTINUE
С
C>-->
         READ AND ECHO TABLE 4. - SPECIFIED CONDITIONS
С
      CALL HEADNG
      PR INT 2200
      IF (KEEP(4) .NE. KEEPI) GO TO 280
      PRINT 2110
         GO TO 330
  280
         J = 1
  290 READ 1060, JSDN(J), <U(J), KV(J), KTH(J), USN(J), VSN(J), THSN(J), IENDN
      IF (IENDN .EQ. IEND) GD TD 300
         J = J + 1
         GD TO 290
         NCT4 = J
  300
  330 PRINT 2210
      DO 335 I = 1, NJ
         IHINGE(I) = ZERO
  335 CONTINUE
      IHINGE(NJ+1) = 0
      DO 420 J = 1, NCT4
         KEY = 2 \neq (2 \neq KU(J) + KV(J)) + KTH(J)
         IF (KEY .GT. 7) KEY = 8
         GO TO (340, 350, 360, 370, 380, 390, 400, 410), KEY
         PRINT 2220, JSDN(J), KU(J), KV(J), KTH(J), THSN(J)
  340
         GO TO 420
         PRINT 2230, JSDN(J), KU(J), KV(J), KTH(J), VSN(J)
  350
         GO TO 420
         PRINT 2230, JSDN(J), KU(J), KV(J), KTH(J), VSN(J), THSN(J)
  360
         GO TO 420
         PRINT 2240, JSDN(J), KU(J), KV(J), KTH(J), USN(J)
  370
         GO TO 420
PRINT 2250, JSDN(J), KU(J), KV(J), KTH(J), USN(J), THSN(J)
  380
         GO TO 420
         PRINT 2240, JSDN(J), KU(J), KV(J), KTH(J), USN(J), VSN(J)
  390
         GO TO 420
         PRINT 2240, JSDN(J), KU(J), KV(J), KTH(J), USN(J), VSN(J), THSN(J)
  400
         GO TO 420
         KEY = JSDN(J)
  410
         IHINGE(KEY) = 1
         PRINT 2260, KEY
  420 CONTINUE
```

```
С
     SUBROUTINE INECHO
                                                                  (CONTINUED)
С
C>-->
         READ AND ECHO TABLE 5. - STATIC LJADING
С
      PRINT 2290
      IF (ISTAT .NE. IYES) GO TO 510
      IF (KEEP(5) .NE. KEEPI) GO TO 430
      PRINT 2110
PRINT 2300
PRINT 2310, (JI5(J), QXI5(J), QYI5(J), JM5(J), QXM5(J), QYM5(J),
     1
                    JL5(J), QXL5(J), QYL5(J), J = 1, NCT5)
      PRINT 2320
         NCI5 = NCT5 + 1
         GO TO 440
         NCI5 = 1
  430
         NCT5 = NCI5
  440
         K = 0
         IND = 0
  450 READ 1080, JI5(NCT5), QXI5(NCT5), QYI5(NCT5), JM5(NCT5), QXM5(NCT5
     1), QYM5(NCT5), JL5(NCT5), QXL5(NCT5), QYL5(NCT5), IENDN
         K = K + 1
         (QXI5(NCT5).EQ.ZERO .AND. QYI5(NCT5).EQ.ZERO) IND = 1
      IF
      IF (IENDN .EQ. IEND) GO TO 460
         NCT5 = NCT5 + 1
         GO TO 450
  460 IF (IND .EQ. 1) GD TD 470
      PRINT 2300
      PRINT 2310, (JI5(J), QXI5(J), QYI5(J), JM5(J), QXM5(J), QYM5(J),
JL5(J), QXL5(J), QYL5(J), J = NCI5, NCT5)
     1
         GO TO 540
  470 IF (K - NCT5) 480, 490, 480
  480 IF (K .EQ. 1) GO TO 500
490 PRINT 2330
      STOP
  500
         NCT5 = NCT5 - 1
  510 PRINT 2340
С
C>-->
         READ AND ECHO TABLE 6. - DYNAMIC LOADING
C
  540 PRINT 2350
         NCL = 0
         (KEEP(6) .EQ. KEEPI) GO TO 550
      IF
         NPDL = 0
         GO TO 570
  550 PRINT 2110
      DO 560 I = 1, NPDL
         NCI = NCL + 1
         NCL = NCL + NCS(I)
         PRINT 2360, 1
         PRINT 2300
         PRINT 2310, (JI6(J), QXI6(J), QYI6(J), JM6(J), QXM5(J),QYM6(J),
                       JL6(J), QXL6(J), QYL6(J), J = NCI, NCL)
     1
  560 CONTINUE
      PRINT 2320
  570 IF (NADL .NE. 0) GD TO 580
      PRINT 2340
         GO TO 630
  580
         NSI = NPDL + 1
         NPDL = NDL
         NS = NSI
          J = NCL
  590
         NCS(NS) = 0
  600
         J = J + 1
      READ 1080, JI6(J), QXI6(J), QYI6(J), JM6(J), QXM6(J), QYM6(J),
                  JL6(J), QXL6(J), QYL6(J), IENDN
     1
         NCS(NS) = NCS(NS) + 1
      IF (IENDN .NE. IEND) GO TO 600
      IF (NS .EQ. NDL) GO TO 510
         NS = NS + 1
         GO TO 590
```

```
SUBROUTINE INECHO
С
                                                                    (CONTINUED)
С
  610
         NCL = 0
      DO 620 I = NSI, NDL
          NCI = NCL + 1
          NCL = NCL + NCS(I)
          PRINT 2360, I
          PRINT 2300
         PRINT 2310, (JI6(J), QXI6(J), QYI6(J), JM6(J), QXM6(J), QYM6(J), JL6(J), QXL6(J), QYL6(J), J = NCI, NCL
     1
  620 CONTINUE
      IF (LDTYPE .EQ. 1) GD TO 630
      READ 1050, FO, TR, TD
      PRINT 2280, TR, TD, FO
С
          READ AND ECHO TABLE 7. - COLLAPSE PARAMETERS
C>-->
C
  630 CALL HEADNG
      PRINT 2370
      IF (NDL .EQ. 0) GO TO 720
IF (KEEP(7) .NE. KEEPI) GJ TJ 640
      PRINT 2110
         GO TO 690
  640 READ 1050, UMAX, VMAX
         NST7 = 1
  650 PEAD 1040, JS7N(NST7), SMAXN(NST7)
      IF (JS7N(NST7) .EQ. NJ) GO TO 660
NST7 = NST7 + 1
          GO TO 650
         NIA7 = 1
  660
  670 READ 1040, JIA7(NIA7), PMULN(NIA7), BMULN(NIA7)
      IF (JIA7(NIA7) .EQ. NJ) GO TO 680
          NIA7 = NIA7 + 1
          GO TO 670
  680 READ 1070, PIAN
      READ 1070, BIAN
  690 PRINT 2380, UMAX, VMAX
PRINT 2390
      DO 700 I = 1, NIA7
          PRINT 2400, JS7N(1), SMAXN(1)
  700 CONTINUE
      PRINT 2410
      DO 710 I = 1, NIA7
         PRINT 2400, JIA7(I), PMULN(I), BMULN(I)
  710 CONTINUE
      PRINT 2420, PIAN
      PRINT 2430, BIAN
          GO TO 730
  720 PRINT 2340
С,
C>-->
          PRINT TABLE 8. - STATION COORDINATES
С
  730 CALL HEADNG
      PRINT 2440
      DO 740 I = 1, NJ
          PRINT 2400, I, X(I), Y(I)
  740 CONTINUE
      RETURN
      END
```

Ĝ

```
SUBROUTINE GEOM
С
C
С
          * * * * * * *
С
Č
C
        THIS SUBROUTINE ESTABLISHES INITIAL GEOMETRY OF THE STRUCTURE
                   JOINT COORDINATES ARE READ IN OR CALCULATED.
Ċ
               INITIAL SLOPES AND LENGTHS OF THE BARS AND INITIAL
                  CURVATURES AT THE JOINTS ARE ALSO DETERMINED
C
C
                         * * * * * * * * * *
С
      SUBROUT INE GEOM (FX)
С
      IMPLICIT REAL * 8(A-H, O-Z)
      COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN
COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE
  COMMON /IGEOM/ PHIO(25), THO(24), IBRK, ISYM
100 FORMAT (10F8.0)
      DATA DM3 / 1.00-03 /
C
         TEST FOR BROKEN LINE STRUCTURE
C>-->
      IF (IBRK .EQ. 1) GO TO 180
C
C>-->
          DIVIDE STRUCTURE IN GIVEN NUMBER DF BARS
         AND CALCULATE JOINT COORDINATES
C
С
      JLAST = NB/2
         TEST FOR SYMMETRICAL STRUCTURE
C>-->
      IF (ISYM .NE. 1) JLAST = NB
      DN = JLAST
      X(1) = ZERO
      Y(1) = ZERO
      XLAST = P5 + SPAN
      IF (ISYM .NE. 1) XLAST = SPAN
      YLAST = FX(XLAST)
      DELS = DSQRT (XLAST*XLAST + YLAST*YLAST)/DN
  110 DJ 140 I = 1, JLAST
         DX = DELS
  120
         XIP1 = X(I) + DX
          YIP1 = FX(XIP1)
          DY = YIP1 - Y(I)
          DXY = DSQRT(DX*DX + DY*DY)
          IF (DABS(DXY-DELS) .LT. DM3) GO TO 130
          DX = DX + DELS / DXY
         GO TO 120
  130
         X(I+1) = XIP1
         Y(I+1) = YIP1
  140 CONTINUE
C
      DX = XLAST - X(JLAST+1)
DY = YLAST - Y(JLAST+1)
      DDELS = DSQRT(DX*DX + DY*DY)
      IF (DDELS .GT. DM3) GO TO 150
         X(JLAST+1) = XLAST
         Y(JLAST+1) = YLAST
         GO TO 160
  150 DDELS = DDELS / DN
      IF (DX .LT. ZERO) DDELS = - DDELS
DELS = DELS + DDELS
      GO TO 110
  160 IF (ISYM .NE. 1) GO TO 190
      DO 170 I = 1, JLAST
         K = NJ - I + 1
         X(K) = SPAN - X(I)
         Y(K) = Y(I)
  170 CONTINUE
      GO TO 190
C
C>-->
         READ JOINT COORDINATES, IF STRUCTURE HAS A BROKEN LINE AXIS
  180 READ 100, (X(I), Y(I), I = 1, NJ)
```

```
SUBROUTINE GEOM
                                                               (CONTINUED)
С
C
         CALCULATE INITIAL LENGTHS AND SLOPES OF THE BARS
c>-->
         AND INITIAL CURVATURES AT THE JOINTS
С
  190 DO 200 I = 1, NB
         DX = X(I+1) - X(I)
         DY = Y(I+1) - Y(I)
         XLO(I) = DSQRT(DX*DX + DY*DY)
         THO(I) = DARSIN(DY/XLO(I))
  200 CONTINUE
      PHIO(1) = ZERO
      DD 210 I = 2, NB
         PHIO(I) = (THO(I) - THO(I-1))/P5/(XLO(I) + XLO(I-1))
  210 CONTINUE
      PHIO(NJ) = ZERO
      RETURN
      END
```

```
С
    SUBROUTINE INTERP
С
С
     *
          * * * * * *
С
     *
С
         THIS SUBROUTINE DISTRIBUTES DATA BY LINEAR INTERPOLATION
С
         BETWEEN STATIONS JS(I-1) TO JS(I), I = 2, NC, FOR NC.GE.2
     *
                                                                    *
С
    *
                                                                    *
          С
     *
С
     SUBROUTINE INTERP (NC, JS, ZN, Z)
С
      IMPLICIT REAL # 8(A-H, O-Z)
      COMMON /CONST/ ZERD, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE
      DIMENSION JS(1), ZN(1), Z(1)
      DO 100 I = 1, NJ
  100 Z(I) = ZERO
      Z(1) = ZN(1)
DO 200 I = 2, NC
        HNEL = JS(I) - JS(I-1)
        DZ = (ZN(I)-ZN(I-1)) / HNEL
        JI = JS(I-1) + 1
        JL = JS(I)
        DO 200 J = JI_{9} JL
  200 Z(J) = Z(J-1) + DZ
      RETURN
      EN D
```

```
FUNCTION FX
C
C
C
C
     *
С
С
С
                    THIS FUNCTION DESCRIBES A HORIZONTAL BEAM,
     *
                                                                              2
                    USED IN PROBLEMS DTP3 AND P2.2 OF SET DTP2
     *
C
C
       FUNCTION FX (X)
С
       IMPLICIT REAL # 8(A-H, O-Z)
      DATA ZERO / 0.0D00 /
               FX = ZERO
       RETURN
       END
```

```
C
     FUNCTION FX
С
C
C
00000
           THIS FUNCTION DESCRIBES A CIRCULAR ARCH HAVING A CENTRAL
           ANGLE OF OPENING OF 87.21 DEGREES, SPAN LO = 687 UNITS OF
     *
                  LENGTH, RISE OF 0.2 LO, USED IN PROBLEM DTP2.1
            * * * *
С
      FUNCTION FX (X)
С
      IMPLICIT REAL * 8(A-H, O-Z)
DATA A, B, C / 841.575D00, 154.575D00, 360.675D00 /
               FX = DSQRT((A-X)*(B+X)) - C
       RETURN
      END
С
     FUNCTION FX
C
C
            * *
                *
                           *
                              *
C C C C C C C
                   THIS FUNCTION DESCRIBES A SEMI-CIRCULAR ARCH
                   HAVING A DIAMETER OF 354.00 UNITS OF LENGTH
                          USED IN PROBLEMS DAP1 AND DAP2
č
       FUNCTION FX (X)
С
       IMPLICIT REAL * 8(A-H, 3-Z)
       DATA D / 354.0000 /
               FX = DSQRT (X + (D-X))
       RETURN
       END
     SUBROUTINE MABC
С
С
Č
            * * * * * * * *
                               *
                                  *
                                    *
                                      ×
                                        *
                                           *
С
С
С
С
             THIS IS A MATRIX MULTIPLICATION ROUTINE. MATRIX A IS
                 MULTIPLIED BY B AND THE RESULT IS STORED IN C
С
                  *
С
       SUBROUTINE MABC (A, B, C, N, M)
C
       IMPLICIT REAL # 8(A-H, D-Z)
       DIMENSION A(N,N), B(N,M), C(N,M)
       DATA ZERO / 0.0000 /
С
               DO 110 I = 1, N
               DO 110 J = 1, M
                     TEMP = ZERO
                     DO 100 K = 1, N
  100
                     TEMP = TEMP + A(I,K) * B(K,J)
                     C(I_{J}) = TEMP
  110
               CONT INUE
               RETURN
                END
```

C SUBROUTINE RTSR 00000000000 THIS IS A MATRIX MULTIPLICATION ROUTINE * * THE SIZE OF ALL MATRICES IS SIX BY SIX THE TRANSPOSE OF R IS MULTIPLIED BY S AND THEN BY R AND THE RESULT IS STORED IN X * * * * * * * * * * * * * * С SUBROUTINE RTSR (R, S, X) С IMPLICIT REAL * 8(A-H, C-Z) DIMENSION R(6,6), S(6,6), X(6,6), T(6,6) DATA ZERO / 0.0D00 / С DO 110 I = 1, 6 DO 110 J = 1, 6 TEMP = ZERODO 100 K = 1, 6 TEMP = TEMP + R(K,I)*S(K,J) 100 T(I,J) = TEMPCONT INUE 110 C CALL MABC (T, R, X, 6, 6) С RETURN END C SUBROUTINE INVERT 0000000000000 THIS SUBROUTINE INVERTS A MATRIX * * INPUT MATRIX IS DESTROYED AND * SUBSTITUTED FOR ITS INVERSE * * * * * * * * * * SUBROUTINE INVERT (X, N, II) С IMPLICIT REAL * 8(A-H, O-Z) DIMENSICN X(N,N) DATA ZERO, EP, ONE / 0.0D00, 1.0D-10, 1.0D00 / С DO 400 I = 1, N IF (DABS(X(I,I)) .LT. EP) GO TO 500 S = ONE / X(I, I)DO 100 J = 1, N 100 X(I,J) = X(I,J) * SX(I,I) = SDO 300 J = 1, NIF (J .EQ. I) 30 TO 300 S = X(J,I) $X(J_{I}) = ZERO$ DO 200 K = 1, N 200 X(J,K) = X(J,K) - S + X(I,K)300 CONTINUE CONT INUE 400 RETURN С 500 PRINT 600, II STOP 600 FORMAT (1H1///, 30X, 27HNO INVERSE EXISTS AT JOINT , 13) END

SUBROUTINE DIST С С С C С THIS SUBROUTINE DISTRIBUTES INPUT DATA TO ALL STATIONS; CALCULATES MASSES AND SELF WEIGHT, INITIAL LOCATIONS OF С CENTROIDS, AXIAL AND FLEXURAL STIFFNESSES AT THE JOINTS C C С С SUBROUTINE DIST c IMPLICIT REAL * 8(A-H, O-Z) Common /arcxy/ x(25), y(25), xLO(24), SPAN COMMON / BMAST/ BMASS(25), BM(25), T(24), SH(24) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE COMMON /CURVS/ EPSN(10,5),SIGN(10,5),EPSMUL(5),SIGMUL(5),EPSPR(5) COMMON / FAIL1/ SMAX(25), SMAXN(10), JS7N(10), UMAX, VMAX, NST7 COMMON /FAIL2/ BMUL(25), PMUL(25), BMULN(10), PMULN(10), PIAN(9), BIAN(9), EPSU(5), JIA7(10), NIA7 1 COMMON /LOADS/ QX(25), QY(25), QZ(25), QXD(25), QYD(25) COMMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9) COMMON /STEEL/ AB(25), AT(25), DB(25), JT(25) COMMON /STIFF/ EI(25), AE(25) COMMON /TABL1/ GRAV,TLIM,KEEP(7),ISTAT,ISOPT,NDL,IDOPT,NOUT,ISELF# COMMON /TABL2/ DN(9,10), B1N(10), B2N(10), B3N(10), DTN(10), ATN(10), 1 DBN(10), ABN(10), JSN(10), JRN(10), NCT2, NRT2 COMMON /TABL5/ JI5(20), QXI5(20), QYI5(20), JM5(20), QXM5(20), QYM5(20), JL5(20), QXL5(20), QYL5(20), NCT5 1 COMMON /TABL6/ JI6(20), QXI5(20), QYI6(20), JM6(20), QXM6(20), QYM6(20), JL6(20), QXL6(20), QYL6(20), NCS(20),NPDL 1 COMMON /XSECT/ B1(25), B2(25), B3(25), CG(25), PC(25) DIMENSION DUM(10), DZON(9) EQUIVALENCE (DUM(1), T(1)), (DZDN(1), T(12)) DATA THREE, SIX, IYES / 3.0D00, 6.0D00, 3HYES / DISTRIBUTE CROSS SECTION DATA C>--> CALL INTERP (NCT2, JSN, BIN, B1) CALL INTERP (NCT2, JSN, B2N, B2) CALL INTERP (NCT2, JSN, B3N, B3) (>-->DISTRIBUTE REINFORCEMENT DATA IF (NRT2 .GT. 1) GO TO BO DO 70 I = 1, NJ AB(I) = ABN(1)AT(I) = ATN(1)DB(I) = DBN(1) $DT(I) \neq DTN(I)$ 70 CONTINUE GO TO 90 80 CALL INTERP (NRT2, JRN, ABN, AB) CALL INTERP (NRT2, JRN, ATN, AT) CALL INTERP (NRT2, JRN, DBN, DB) CALL INTERP (NRT2, JRN, DTN, DT) DISTRIBUTE ZONE DEPTHS (>--> 90 DO 120 I = 1, 9 DO 100 J = 1, NCT2 DUM(J) = DN(I,J)100 CALL INTERP (NCT2, JSN, DUM, QZ) D3 110 J = 1, NJ D(J,I) = QZ(J)110 120 CONTINUE С C>--> DISTRIBUTE MASS AND SELF-WEIGHT DO 130 I = 1, NJ QX(I) = ZEROOY(I) = ZEROQZ(I) = ZERO130 CONTINUE I = 1DO 135 J = 1, 9135 DZON(J) = D(I,J)

```
SUBROUTINE DIST
                                                                  (CONTINUED)
С
      CALL MASS (DZON, B1(I), B2(I), B3(I), DB(I), DT(I), AB(I),
     1
                  AT(I). AGAM)
         AGL = AGAM * XLO(I)
      QY(I) = QY(I) - AGL / THREE
      QY(I+1) = QY(I+1) - AGL / SIX
      DO 140 I = 2, NB
         DO 138 J = 1, 9
  138
         DZON(J) = D(I,J)
      CALL MASS (DZON, B1(I), B2(I), B3(I), DB(I), DT(I), AB(I),
     ł
                  AT(I), AGAM)
         AGL = AGAM + XLO(I-1)
         QY(I-1) = QY(I-1) - AGL/SIX
          QY(I) = QY(I) - AGL/THREE
         AGL = AGAM + XLO(I)
          QY(I) = QY(I) - AGL/THREE
         QY(I+1) = QY(I+1) - AGL/SIX
  140 CONTINUE
      I = NJ
      DO 145 J = 1, 9
  145 DZON(J) = D(I,J)
      CALL MASS (DZON, B1(I), B2(I), B3(I), DB(I), DT(I), AB(I),
     1
                  AT(I), AGAM)
          AGL = AGAM + XLO(NB)
      QY(I-1) = QY(I-1) - AGL / SIX
      QY(I) = QY(I) - AGL/THREE
SELFW = ISELFW
      D3 150 I = 1, NJ
         BMASS(I) = -QY(I) / GRAV
         QY(I) = SELFW + QY(I)
  150 CONTINUE
С
         DISTRIBUTE STATIC LOAD DATA
C>-->
      IF (ISTAT .NE. IYES) GO TO 160
      CALL DFORCE (NCT5, JI5, QXI5, QYI5, JM5, QXM5, QYM5,
     1
                           JL5, QXL5, QYL5, QX, QY)
С
C>-->
         CALCULATE INITIAL LOCATION OF CENTROID AT EACH
         CROSS SECTION, AXIAL AND FLEXURAL STIFFNESSES
С
  160 DJ 170 J = 1, NSSC
         E(J,1) = SIGMUL(J) + SIGN(5,J) / (EPSMUL(J) + EPSN(5,J))
         E(J_{2}) = SIGMUL(J) * SIGN(5,J) / (EPSMJL(J) * EPSN(6,J))
  170 CONTINUE
      DO 190 I = 1, NJ
         DO 180 J = 1, 9
         DZON(J) = D(I,J)
  180
         CALL CENTER (DZON, B1(I), B2(I), B3(I), AT(I), DT(I),
                       AB(I), DB(I), PC(I), AE(I), EI(I))
     ł
         CG(I) = PC(I)
  190 CONTINUE
C
C>-->
         DISTRIBUTE FAILURE PARAMETERS
      IF (NDL .EQ. 0) RETURN
      IF (NST7 .GT. 1) GD TO 210
      DO 200 I = 1, NJ
         SMAX(I) = SMAXN(1)
  200 CONTINUE
      GO TO 220
  210 CALL INTERP (NST7, JS7N, SMAXN, SMAX)
  220 IF (NIA7 .GT. 1) GO TO 240
      DO 230 I = 1, NJ
PMUL(I) = PMULN(1)
         BMUL(I) = BMULN(1)
  230 CONTINUE
      RETURN
  240 CALL INTERP (NIA7, JIA7, PMULN, PMUL)
Call Interp (NIA7, JIA7, BMJLN, BMUL)
      RETURN
      EN D
```

```
С
      SUBROUTINE DFORCE
С
С
С
               THIS SUBROUTINE DISTRIBUTES STATIC OR DYNAMIC LOADING
AS CONCENTRATED LOADS AT SPECIFIED STATIONS
C
С
Č
                С
        SUBROUTINE DFORCE (NC, JI,QIX,QIY, JM,QMX,QMY, JL,QLX,QLY, QX, QY)
С
       IMPLICIT REAL * 8(A-H, 0-Z)
COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN
       COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE
DIMENSICN JI(1), QIX(1), QIY(1), JM(1), QMX(1), QMY(1),
JL(1), QLX(1), QLY(1), QX(1), QY(1)
      1
С
        DO 180 J = 1, NC
С
           DETERMINE TYPE OF SPECIFIED LOADING
C>-->
           IF (QIX(J).EQ.ZERO .AND. QIY(J).EQ.ZERO) GO TO 160
IF (QMX(J).EQ.ZERO .AND. QMY(J).EQ.ZERO) GO TO 140
IF (QLX(J).EQ.ZERO .AND. QLY(J).EQ.ZERO) GO TO 120
С
           PARABOLIC DISTRIBUTION FROM JI(J) TO JL(J)
C>-->
           IF (QIX(J) .NE. ZERO)
           CALL DPARAB (JI(J), JM(J), JL(J), QIX(J), QMX(J), QLX(J), Y, QX)
      1
           IF (QIY(J) .NE. ZERO)
           CALL DPARAB (JI(J), JM(J), JL(J), QIY(J), QNY(J), QLY(J), X, QY)
      1
           GO TO 180
С
           LINEAR DISTRIBUTION FROM JI(J) TO JM(J)
C>-->
           IF (QIX(J) .NE. ZERG)
CALL LINEAR (JI(J), JM(J), QIX(J), QMX(J), Y, QX)
  120
      1
           TF (QIY(J) .NE. ZERO)
           CALL LINEAR (JI(J), JM(J), QIY(J), QMY(J), X, QY)
      1
           GO TO 180
           IF (QLX(J) .EQ. ZERO) GO TO 150
  140
C
C>-->
           PRESSURE DISTRIBUTION FROM JI(J) TO JL(J)
           JIJ = JI(J) + 1JLJ = JL(J)
           DO 145 I = JIJ, JLJ
                  DX = X(I) - X(I-1) 
DY = Y(I) - Y(I-1)
                  DQX = P5 * QIX(J) * DY
DQY = P5 * QUE(J) * DX
                  QX(I) = QX(I) - DQX
                  QX(I-1) = QX(I-1) - DQX
                  QY(I) = QY(I) + DQY
                  QY(I-1) = QY(I-1) + DQY
           CONTINUE
  145
           GO TO 180
С
C>-->
           CONCENTRATED LOAD AT JI(J)
           JIJ = JI(J)
  150
           QX(JIJ) = QX(JIJ) \rightarrow QIX(J)
           QY(JIJ) = QY(JIJ) + QIY(J)
           GO TO 180
C
           PRINT MESSAGE FOR UNIDENTIFIABLE TYPE OF LOADING AND STOP
C>-->
C
           PRINT 170, JI(J)
Format ( ///30x,33HUNIDENTIFIABLE LOAD TYPE AT JOINT, I5,
  160
  170
                    //30X, 21HPROGRAM DISCONTINUED.)
      1
           STOP
  180 CONTINUE
C
        RETURN
        EN D
```

SUBROUTINE DPARAB С С 000000 THIS SUBROUTINE DISTRIBUTES SPECIFIED PARABOLIC LOADING AS CONCENTRATED LOADS FROM STATIONS JI TO JL * * * -* * * * * ± . Ĉ SUBROUTINE DPARAB (JI, JM, JL, QIN, QMN, QLN, Z, Q) С IMPLICIT REAL * 8(A-H, J-Z) DIMENSION Z(1), Q(1) DATA TWO, FOUR, TWEL / 2.0000, 4.0000, 1.2001 / С Q1 = QINQ2 = QMNZ1 = Z(JI)Z2 = Z(JM)Z3 = Z(JL)FAC1 = (Z2-Z1) * (Z3-Z1)FAC2 = (Z2-Z1) * (Z2-Z3)FAC3 = (Z3-Z1) * (Z3-Z2)FAC1 = Q1 / FAC1 FAC2 = Q2 / FAC2FAC3 = QLN/ FAC3 A = FAC1 + FAC2 + FAC3JIP1 = JI + 1DO 100 J = JIP1, JL Q2 = FAC1 * (Z(J)-Z3) * (Z(J)-Z2) + FAC2 + (Z(J)-Z1) + (Z(J)-Z3) +1 FAC3 + (Z(J)-Z2) + (Z(J)-Z1)2 DZ = DABS(Z(J)-Z(J-1))Q(J-1) = Q(J-1) + DZ * (FOUR * Q1 + TWO* Q2 - A* DZ * DZ) / TWEL Q(J) = Q(J) + DZ*(T#J*Q1 + FOUR*Q2 - A*DZ*DZ) / TWEL 100 Q1 = Q2RETURN END 000000000 SUBROUTINE LINEAR THIS SUBROUTINE DISTRIBUTES SPECIFIED LINEAR LOADING * AS CONCENTRATED LOADS FROM STATIONS JI TO JL * С SUBRDUTINE LINEAR (JI, JL, QI, QL, Z, Q) С IMPLICIT REAL * 8(A-H, O-Z) DIMENSION Z(1), Q(1) DATA TWO, THREE, SIX / 2.0000, 3.0000, 6.0000 / C QLEFT = QI DELQ = (QL-QI) / DABS(Z(JL)-Z(JI))JIPI = JI + 1 $00 \ 100 \ I = JIP1, JL$ DZ = DABS(Z(I)-Z(I-1))DQ = DZ + DELQQ(I-1) = Q(I-1) + DZ + (THREE*QLEFT + DQ) / SIXQ(I) = Q(I) + DZ + (THREE*QLEFT + TWO*DQ) / SIX100 QLEFT = QLEFT + DQRETURN END

```
SUBROUTINE CENTER
С
С
C
С
С
           THIS SUBROUTINE CALCULATES INITIAL LOCATION OF CENTROID,
            AXIAL AND FLEXURAL STIFFNESSES AT GIVEN CROSS SECTION
C
С
С
               С
      SUBROUT INE CENTER (DZON, B1, B2, B3, AT, DT, AB, DB, DBAR, SAE, SEI)
С
      IMPLICIT REAL * 8(A-H, O-Z)
      COMMON /CONST/ ZERD, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE
      COMMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9)
      COMMON /SEGM2/ DI(30), DA(30), MI(30)
      DIMENSION DZON(1)
C
         CALCULATE TRANSFORMED AREA OF THE SECTION
C>-->
         AND FIRST MOMENT ABOUT THE TOP
C
      SAE = ZERO
      SEI = ZERO
      SDAE = ZERO
      CALL SEG (DZON, B1, B2, B3)
      DO 100 J = 1, 30
         MM = MI(J)
         DAE = DA(J) + E(MM_{1})
         SAE = SAE + DAE
         SDAE = SDAE + DAE*DI(J)
  100 CONTINUE
         TEST FOR TOP REINFORCEMENT
C>-->
      IF (AT .EQ. ZERO) GO TO 130
         DETERMINE TYPE OF MATERIAL AT THE LEVEL OF TOP REINFORCEMENT
C>-->
      DO 110 J = 1, 30
         IF (DT .GT. DI(J)) GO TO 110
         MT = MI(J)
         GO TO 120
  110 CONTINUE
C>-->
         ADD CONTRIBUTION OF TOP REINFORCEMENT
  120 \text{ DAE} = \text{AT} = (E(\text{NSSC}, 1) - E(\text{MT}, 1))
      SAE = SAE + DAE
      SDAE = SDAE + DAE + DT
         TEST FOR BOTTOM REINFORCEMENT
C>-->
  130 IF (AB .EQ. ZERO) GO TO 160
         DETERMINE TYPE OF MATERIAL AT THE LEVEL OF BOTTOM REINFORCEMENT
C>-->
      DO 140 J = 1, 30
         K = 31 - J
         IF (DB .LT. DI(K)) GD TO 140
         MB = MI(K)
         GO TO 150
  140 CONTINUE
         ADD CONTRIBUTION OF BOTTOM REINFORCEMENT
C>-->
  150 \text{ DAE} = \text{AB} * (E(NSSC_{1}) - E(MB_{1}))
      SAE = SAE + DAE
      SDAE = SDAE + DAE + DAE
         CALCULATE DEPTH OF CENTROID
C>-->
  160 DBAR = SDAE / SAE
         CALCULATE AXIAL AND FLEXJRAL STIFFNESSES
C>-->
      DD 170 J = 1, 30
         MM = MI(J)
         DJ = DI(J) - DBAR
         SEI = SEI + DA(J) \neq E(MM, 1) \neq DJ \neq DJ
  170 CONTINUE
      IF (AT .EQ. ZERO) GO TO 180
DJ = DT - DBAR
      SEI = SEI + AT * (E(NSSC,1)-E(MT,1)) * DJ * DJ
  180 IF (AB .EQ. ZERO) RETURN
      DJ = DB - DBAR
      SEI = SEI + AB * (E(NSSC, 1)-E(MB, 1)) * DJ * DJ
      RETURN
      END
```
С SUBROUTINE MSTIF C Ċ 000000 ٠ * THIS SUBROUTINE CALCULATES MEMBER STIFFNESSES IN LOCAL COORDINATES AND SETS UP TRANSFORMATION * * MATRIX TO GLOBAL COORDINATE SYSTEM * * * * * * * * * * * * * * * * * * * C SUBROUTINE MSTIF (DELS, IND, EIZ, AEX, COST, SINT, S, R) С IMPLICIT REAL * 8(A-H, 'O-Z) DIMENSION S(6,6), R(6,6) DATA ZERD, ONE, TWD, SIX / 0.0D00, 1.0D00, 2.0D00, 6.0D00 / DATA THREE, FOUR, TWEL / 3.0000, 4.0000, 1.2001 / С DO 100 I = 1, 6 DO 100 J = 1, 6 S(I,J) = ZEROR(I,J) = ZERO100 CONTINUE GO TO (110, 120, 130), IND C22 = THREEC26 = THREE110 C33 = ZERO C36 = ZEROC66 = THREE GD TO 140 C22 = THREE 120 C26 = ZEROC33 = THREEC36 = ZEROC66 = ZEROGO TO 140 130 C22 = TWELC26 = SIXC33 = FOUR C36 = TWOC66 = FOUR140 AEOL = AEX / DELS EIOL = EIZ / DELS EIOL2 = EIOL / DELS EIOL3 = EIOL2 / DELS S(1,1) = AEOLS(1,4) = - AEOLS(4,4) = AEDLS(2,2) = C22 * EIOL3S(2,3) = (IND-1) * THREE * EIJL2 S(2,5) = - S(2,2)S(2,6) = C26 = EIOL2S(3,3) = C33 * EIDL S(3,5) = - S(2,3)S(3,6) = C36 * EIOL S(5,5) = S(2,2)S(5,6) = - S(2,6)S(6,6) = C66 + EIOLS(4,1) = S(1,4)DO 150 I = 2, 5 DO 150 J = I, 6 S(J,I) = S(I,J)CONTINUE 150 IF (IND .EQ. 3) GO TO 160 S(3*IND, 3*IND) = ONER(1,1) = COST160 R(2,2) = COSTR(1,2) = SINTR(2,1) = - SINTR(3,3) = ONER(4,4) = COST

с с

R(5,5)	æ	COST
R(6,6)	Ŧ	ONE
R(4,5)	æ	SINT
R(5,4)	=	- SINT
RETURN		
END		

c		SUBROUTINE SEG
ĉ		* * * * * * * * * * * * * * * * * * * *
		* THIS SUBROUTINE DEFINES TYPE OF MATERIAL AND * * CALCULATES AREAS AND DEPTHS OF CENTROIDS * * OF EACH SEGMENT IN GIVEN SECTION *
č		* * * * * * * * * * * * * * * * * * * *
с с		SUBROUTINE SEG (DZON, B1, B2, B3)
ſ		IMPLICIT REAL * 8(A-H, O-Z) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE COMMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9) COMMON /SEGM2/ DI(30), DA(30), MI(30) DIMENSION DZON(1)
Ŭ		DO 200 I = 1, 9
	100	GO TO (100, 150, 110, 120, 150, 150, 130, 140, 150), I DTOP = ZERO BTOP = B1 BBOT = B1 ISTRT = 1
		ITOP = 0
		GO TO 160
	110	BBOT = B2 GO TO 150
	120	BTOP = B2
	130	BBOT = B3
		GO TO 150
	140	BTOP = B3
	190	DTOP = 15(1-1) DTOP = DZON(1-1)
		ISTRT = IS(I-1) + 1
	160	HN = IS(I) - ITOP
		IF (DZON(I) .LE. DTOP) GO TO 170
		DELB = (BTOP - BBJT) / (DZON(I) - DTOP)
		GO TO 180
	170	HN = ZERO
	1 8 4	DELB = ZERO
	100	I STOP = IS(I)
		IF (ISTRT .GT. ISTOP) GO TO 200
		DO 190 J = ISTRT, ISTOP
		DA(J) = HN + (BIUP - PS+DELB+HN) DI(J) = DTOP + PS+HN
		MI(J) = M
		DTOP = DTOP + HN
	100	BTOP = BTOP - DELB*HN
	200	CONTINUE
	200	RETURN
		END

(CONTINUED)

97

C	S	UBROUTINE STATIC	
C		* * * * * * * * * * * * * * * * * * * *	*
Č			•
C C	4	THIS SUBROUTINE CONTRULS THE STATIC SULUTION BY CALLING OTHER SUBROUTINES, IT SOLVES FOR	* *
č	4	STATIC DISPLACEMENTS, REFINES THE SOLUTION,	*
C	4	SOLVES FOR INTERNAL FORCES AND REACTIONS	*
č			*
C	1	* * * * * * * * * * * * * * * * * * * *	*
L		SUBROUTINE STATIC	
C			
		IMPLICIT REAL * 8(A-H, U-Z) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYP	E
		COMMON /DISPL/ U(25), V(25), TH(25), UD(25), VD(25)	- .
		COMMON /EPSOY/ XM1(25,32), XM2(25,32), XM3(25,32), ICRAC(25,32) COMMON /ICADS/ OX(25), OX(25), O7(25), OXD(25), OVD(25)	
•		COMMON /TABL1/ GRAV,TLIM,KEEP(7),ISTAT,ISOPT,NDL,IDOPT,NOUT,ISELF	N
		DIMENSION US (25, 3), UA (25, 3), FS (25, 3), F(25, 3), RES (25, 3)	
	1	(FS(1,1),XM3(1,1)), (F(1,1),XM3(1,17)),	
	, a	(RES(1,1), XM3(1,9))	
	1	28HCONVERGE AFTER 10 ITERATIONS)	
		DATA DM5 / 1.0 D-05 /	
L		DO 130 I = 1. NJ	
		FS(1,1) = QX(1)	
		FS(1+2) = QY(1) FS(1-3) = Q7(1)	
		130 J = 1, 3	
	120	F(I,J) = FS(I,J)	
	100	CALL SOLVE	
		$\begin{array}{cccccccccccccccccccccccccccccccccccc$	
		US(1,J) = UA(1,J)	
	140	CONTINUE	
	150	ITER = ITER + 1	
		CALL RESIDU	
		DU 160 I = 1, NJ DU 160 J = 1, 3	
		IF (DABS(RES(I,J)) .GT. DN5) GO TO 170	
	160	GO TO 200	
	170	DO 180 I = 1, NJ	
		DO 180 J = 1, 3 $F(T_{0,1}) = RFS(T_{0,1})$	
	180	CONTINUE	
		CALL SOLVE	
		D0 190 J = 1, 3	
	100	US(I,J) = US(I,J) + UA(I,J)	
	190	IF (ITER +LT. 10) GO TO 150	
		PRINT 1	
	200	510° DO 210 I = 1 + NJ	
		U(1) = US(1,1)	
		V(1) = US(1,2) TH(1) = US(1.3)	
	210	CONTINUE	
		CALL STEOR CALL DUTPUT (ISOPT, ZERO, U. V)	
		RETURN	
		END	

SUBROUTINE SOLVE C C С С С С ź THIS SUBROUTINE SOLVES FOR STATIC DISPLACEMENTS USING RECURSION-INVERSION ALGORITHM, BASED ON ± 0 0 0 0 0 GAUSS ELIMINATION FOR BANDED MATRICES * SUBROUTINE SOLVE С IMPLICIT REAL * 8(A-H, O-Z) COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN COMMON /CUNST/ ZERO, P5, DNE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE COMMON /EPSOY/ XM1(25,32), XM2(25,32), XM3(25,32), ICRAC(25,32) COMMON /STIFF/ EI(25), AE(25) COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26), JSDN(10), KU(10), KV(10), KTH(10), NCT4 1 DIMENSION A(26,3), B(26,3,3), UA(25,3), F(25,3) DIMENSION AA(3,3), BB(3,3), CC(3,3), DD(3,1),S(6,6),R(6,6),ST(6,6) DIMENSION AM1(3,1), BM1(3,3), ATM(3,1), BTM(3,3), UP1(3,1) EQUIVALENCE (A(1,1), XM1(1,17)), (B(1,1,1), XM2(1,17)), (UA(1,1), XM2(1,1)), (F(1,1), XM3(1,17)), 1 (XM3(1,25), S), (XM3(1,27), R), (XM3(1,29), ST) 2 C $DO \ 100 \ I = 1, 6$ DO 100 J = 1, 6S(I,J) = ZEROCONTINUE 100 DO 120 J = 1, 3A(1,J) = ZERDDO 110 K = 1, 3110 B(1, J, K) = ZEROCONT INUE 120 С C>--> START RECURSION PROCESS DO 300 I = 1, NJ $DO \ 130 \ J = 1, 3$ DO 130 K = 1, 3 $AA(J_{\tau}K) = S(J+3_{\tau}K)$ BB(J,K) = S(J+3, K+3)130 CONTINUE IND = 3IF (IHINGE(I) .EQ. 1) IND = 1 IF (IHINGE(I+1) .EQ. 1) IND = 2 IEQ = 8DO 140 J = 1, NCT4IF (JSDN(J) .NE. I) GD TO 140 KEY = 2 * (2*KU(J) + KV(J)) + KTH(J) IF (KEY .GT. 7) GO TO 140 IN = JIEQ = KEY 140 CONTINUE IF (I .LT. NJ) GO TJ 160 DO 150 J = 1, 6DO 150 K = 1, 6 S(J,K) = ZERD CONTINUE 150 GO TO 166 DX = X(I+1) - X(I)160 DY = Y(I+1) - Y(I)CX = DX / XLO(I)CY = DY / XLO(I)EIZ = P5 * (EI(I)+EI(I+1))AEX = P5 * (AE(I)+AE(I+1))CALL MSTIF (XLO(I), IND, EIZ, AEX, CX, CY, ST, R) CALL RTSR (R, ST, S) 166 DO 180 J = 1, 3 $AM1(J_1) = A(I_J)$

98

C C	1	SUBROUTINE SOLVE		. ((CONTINUED)
•		DO 170 $< = 1, 3$			
		$BB(J_*K) = BB(J_*K) + S(J_*K)$			
		CC(J,K) = S(J, K+3)			
	170	$BM1(J_{1}K) = B(I_{1}J_{1}K)$			
	180	CONTINUE			
		DD(1,1) = -F(1,1)			
		DD(2+1) = -F(1+2)			
		DD(3,1) = -F(1,3)			
		GO TO (240, 210, 210, 190, 190, 190,	190,	260),	IEQ
	190	$DO \ 200 \ J = 1, 3$		1	
		AA(1,J) = ZERO			
		BB(1,J) = ZERO			
		CC(1,J) = ZERU			
	200				
		BB(1+1) = UNE			
		CO TO (240, 210, 210, 260, 240, 210, 210, 210, 210, 210, 210, 210, 21	210.	2601	150
	210	$\frac{10}{220} = \frac{1}{2} = \frac{3}{20}$	2109	20019	1 CM
	210	$\Delta A(2.1) = 7 \text{ FR} \Omega$			
•		BB(2.1) = ZERO			
		CC(2,J) = ZERO			
	220	CONTINUE			
		BB(2,2) = ONE			
		DD(2+1) = -VSN(IN)			
		GO TO (240, 260, 240, 250, 240, 260,	240,	250),	IEQ
	240	DO 250 J = 1, 3			
		AA(3,J) = ZERO			
		BB(3,J) = ZERO			
		CC(3,J) = ZERO			
	250	CONTINUE			
		$BB(3_13) = UNE$			
	240	UU(3+1) = - (HSV(1N))			
	200	CALL MADE (AA) DML9 D(M) J9 J9 Call MARC (AA, AM1, ATM, 3, 1)			
		CALL MADE (AA) AMI) AIM) DI II Do 270 l'= 1, 2			
		$\Delta TM(J_{\bullet}1) = \Delta TM(J_{\bullet}1) + DD(J_{\bullet}1)$		•	
		DO 270 K = 1.3			
		$BTM(J_*K) = BTM(J_*K) + BB(J_*K)$			
	270	CONTINUE			
		CALL INVERT (BTM, 3, I)			
		CALL MABC (BTM, ATM, AM1, 3, 1)	•		
		CALL MABC (BTM, CC, BB, 3, 3)			
		DO 290 J = 1, 3			
		A(I+1,J) = -AM1(J,1)			
		DO 280 K = 1, 3			
	280	$B(I+I_{1}J_{1}K) = -BB(J_{1}K)$			
	290				
c	300	CUNTINUE			
		STADT DEVERSION DROCESS (RACKSURSTITITION)			
67	/	$\frac{310}{10} = 1.3$			
		(UP1(.1,1) = A(N.1+1,.1))			
		UA(NJ,J) = UP1(J,J)			
	310	CONT INUE			
		DD 340 I = 1, NB			
		J = NJ - I			
		DO 320 K = 1, 3			
•		DO 320 L = 1, 3			
		BTM(K,L) = B(J+1,K,L)			
	320	CONT INUE			
		CALL MABC (BTM, UP1, ATM, 3, 1)			
		$\begin{array}{c} UU \ 3 \ 3 \ 0 \ 1 \ 1 \ 1 \ 1 \ 1 \ 1 \ 1 \ 1 \ 1$			
		$\bigcup_{X \in [X]} = \bigcup_{X \in [X]} = \bigcup_{X \in [X]} + A(J+1)K$			
	220	CONTINUE			
	340				
	J+U	RETURN			
		END			

c s	SUBROUTINE RESIDU											
	* * * * * * * * * * * * * * * * * * * *											
C I	THIS SUBROUTINE CALCULATES THE RESIDUES *											
C +	CF THE SYSTEM OF SIMULTANEOUS EQUATIONS * USED IN THE STATIC SOLUTION *											
C												
С ч С												
с	SUBROUT INE RESIDU											
•	IMPLICIT REAL * 8(A-H, O-Z)											
	COMMON /ARCXY/ X(25), Y(25), XLJ(24), SPAN COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE											
	COMMON /EPSOY/ XM1(25,32), XM2(25,32), XM3(25,32), ICRAC(25,32) COMMON (STIEE/ E1(25), AE(25))											
	COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26),											
1	L JSDN(10), KU(10), KV(10), KTH(10), NCT4 DIMENSION AA(3.3), BB(3.3), CC(3.3), DD(3.1), AUX(3.1), AUXU(3.1)											
	DIMENSION US(25,3), FS(25,3), RES(25,3), S(6,5), R(6,6), ST(6,6)											
1	$\begin{array}{c} \text{EQUIVALENCE (US(1,1), XM1(1,1)), (FS(1,1), XM3(1,1)),} \\ \text{(RES(1,1), XM3(1,9)), (XM3(1,25), S(1,1)),} \end{array}$											
<u>د</u>	2 (XM3(1,27), R(1,1)), (XM3(1,29), ST(1,1))											
C>>	INITIALIZE AUXILIARY MATRICES											
C	DO 100 I = 1, 3											
	$AUXU(I_{1}) = ZERO$											
	AA(I,J) = ZERO											
1.00	BB(I,J) = ZERO CONTINUE											
100	DO 300 I = 1, NJ											
с	IND v≖ 3											
C>>	SEARCH FOR HINGES											
L	IF (IHINGE(I) \cdot EQ. 1) IND = 1											
	IF (IHINGE(I+1) .EQ. 1) IND = 2 IEO = 8											
	DO 130 J = 1, NC T4											
	1F (JSDN(J) •NE• 1) GU IJ 130 KEY = 2 * (2*KU(J) + KV(J)) + KTH(J)											
	IF(KEY .GT. 7) GO TO 130											
•	IJ = J											
130	CONTINUE											
č>>	DETERMINE MEMBER STIFFNESS MATRIX IN GLOBAL COORDINATE SYSTEM											
C	IF (I .LT. NJ) GO TO 150											
	DO 140 J = 1, 6											
	S(J,K) = ZERO											
140												
150	DX = X(I+1) - X(I)											
•	DY = Y(I+1) - Y(I) $CX = DX / XLQ(I)$											
	CY = DY / XLO(I)											
	EIZ = P5 + (EI(I)+EI(I+1)) AEX = P5 + (AE(I)+AE(I+1))											
	CALL MSTIF (XLO(I), IND, EIZ, AEX, CX, CY, ST, R)											
160	$\begin{array}{l} \text{CALL KISK (K, SI, S)} \\ \text{DO 170 J = 1, 3} \end{array}$											
	DO 170 K = 1, 3 BR(L,K) = BR(L,K) + S(L,K)											
-	CC (J+K) = S(J+K+3)											
170	CONTINUE											

С SUBROUTINE RESIDU (CONTINUED) С с> с SET UP LOAD FUNCTIONS DD(1,1) = -FS(1,1)DD(2,1) = -FS(1,2)DD(3,1) = -FS(1,3)С c>--> CONSIDER SPECIFIED DISPLACEMENT CONDITIONS GO TO (230, 200, 200, 180, 180, 180, 180, 250), IEQ С <>--> SPECIFIED HORIZONTAL DISPLACEMENT 180 DO 190 J = 1, 3 AA(1, J) = ZEROBB(1,J) = ZERQCC(1,J) = ZERO190 CONTINUE BB(1,1) = ONEDD(1,1) = - USN(1J)GO TO (230, 200, 200, 250, 230, 200, 200, 250), IEQ C C>--> SPECIFIED VERTICAL DISPLACEMENT 200 $00\ 210\ J = 1,\ 3$ AA(2,J) = ZEROBB(2, j) = ZEROCC(2,J) = ZEROCONTINUE 210 BB(2,2) = ONEDD(2,1) = -VSN(IJ)GO TO (230, 250, 230, 250, 230, 250, 230, 250), IEQ С C>--> SPECIFIED ROTATION DO 240 J = 1, 3 230 AA(3,J) = ZEROBB(3,J) = ZEROCC(3,J) = ZEROCONT INUE 240 BB(3,3) = ONEDD(3,1) = - THSN(IJ)¢ C>---> CALCULATE RESIDUES C CALL MABC (AA, AUXU, AUX, 3, 1) DO 260 J = 1, 3 250 RES(I,J) = -DD(J,1) - AUX(J,1)AUXU(J,1) = US(I,J) 260 CALL MABC (BB, AUXU, AUX, 3, 1) DO 270 J = 1, 3 RES(I,J) = RES(I,J) - AJX(J,1) 270 IF (I .EQ. NJ) 30 TO 300 DO 280 J = 1, 3280 $AUXU(J_{1}) = US(I+1_{J})$ CALL MABC (CC, AUXU, AUX, 3, 1) DO 290 J = 1, 3RES(I,J) = RES(I,J) - AUX(J,1)AUXU(J,1) = US(I,J)DO 290 K = 1, 3 $A\overline{A}(J,K) = S(J+3, K)$ BB(J,K) = S(J+3, K+3)290 CONT INUE CONTINUE 300 RETURN END

C SUBROUTINE STEOR С Ĉ С C THIS SUBROUTINE CALCULATES STATIC INTERNAL FORCES AND REACTIONS. IT ALSO PRINTS DUT C ÷ Ċ STATIC DISPLACEMENTS AND REACTIONS Ċ * * * * * * * * * * * * C SUBROUT INE STFOR С IMPLICIT REAL * 8(A-H, O-Z) COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN COMMON /BMAST/ BMASS(25), BM(25), T(24), SH(24) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE COMMON /EPSOY/ XM1(25,32), XM2(25,32), XM3(25,32), ICRAC(25,32) COMMON /STIFF/ EI(25), AE(25) CDMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26), JSDN(10), KU(10), KV(10), KTH(10), NCT4 1 DIMENSION US(25,3), FS(25,3), R(10,3), JR(10), S(6,6), RT(6,6) DIMENSION UT(6,1), FT(6,1), FL(3), FR(3), RD(2) EQUIVALENCE (US(1,1), XM1(1,1)), (FS(1,1), XM3(1,1)), (XM3(1,25), S), (XM3(1,27), RT), (XM3(1,29), R) 1 2020 FDRMAT (//20X, 31HTABLE 9. - STATIC DISPLACEMENTS//30X, 1 4HSTA., 12X, 1HJ, 11X, 1HV, 11X, 2HTH/) 2030 FDRMAT (30X, 14, 6X, 1P3D12.4) 2100 FORMAT (//20X, 28HTABLE 10. - STATIC REACTIONS//30X, 4HSTA., 8X, 33HH OR I ZONTAL VERTICAL MOMENT/) 1 С PRINT STATIC DISPLACEMENTS <>--> CALL HEADNG **PRINT 2020** DO 80 I = 1, NJ 80 PRINT 2030, I, (US(I,J), J= 1, 3) INITIALIZE AUXILIARY VECTORS AND REACTIONS c>--> $RD(1) \neq ZERO$ RD(2) = ZERODO 100 I = 1, 10 JR(I) = 0DO' 100 J = 1, 3 $R(I_{+}J) = ZERO$ 100 CONTINUE M = 0DO 200 I = 1, NBIND = 3SEARCH FOR HINGES C>--> IF (IHINGE(I) .EQ. 1) IND = 1IF (IHINGE(I+1) .EQ. 1) IND = 2 c>--> CALCULATE MEMBER STIFFNESSES DX = X(I+1) - X(I)DY = Y(I+1) - Y(I)CX = DX / XLO(I) CY = DY / XLO(I)EIZ = P5 * (EI(I) + EI(I+1))AEX = P5 * (AE(I)+AE(I+1))CALL MSTIF (XLO(I), IND, EIZ, AEX, CX, CY, S, RT) CALCULATE INTERNAL FORCES C>--> DO 130 J = 1, 3 FT(J,1) = US(I,J)FT(J+3,1) = US(I+1,J)130 CONT INUE CALL MABC (RT, FT, UT, 6, 1) CALL MABC (S, UT, FT, 6, 1) FL(1) = - FT(1,1)FL(2) = FT(2,1)FL(3) = - FT(3,1)FR(1) = FT(4,1)FR(2) = - FT(5,1)FR(3) =FT(6,1)

SUBROUTINE STFOR (CONTINUED) С С C>--> CALCULATE STATIC BENDING MOMENTS, THRUSTS AND SHEARS С BM(I) = FL(3)T(I) = FR(I)SH(I) = FL(2)С C>--> SEARCH FOR SPECIFIED CONDITIONS С DO 190 J = 1, NCT4 IF (JSDN(J) .NE. I) GO TO 190 K = 2 * (2*KU(J) + KV(J)) + KTH(J) IF (K .GT. 7) GD TD 190 M = M + 1JR(M) = JSDN(J)M1 = (K-1) * (K-2) * (K-3)IF (M1 .EQ. 0) GO TO 140 R(M,1) = -FS(1,1) + RD(1) + FT(1,1) + CX - FT(2,1) + CYGO TO 150 140 $R(M_{1}) = ZERO$ M2 = (K-1) * (K-4) * (K-5)150 IF (M2 .EQ. 0) GO TO 160 R(M,2) = -FS(I,2) + RD(2) + FT(1,1)*CY + FT(2,1)*CXGO TO 170 160 R(M,2) = ZERO٠. M3 = (K-2) * (K-4) * (K-6)170 IF (M3 .EQ. 0) GO TO 180 R(M,3) = -FS(I,3) + FT(3,1)GO TO 190 $R(M_{3}) = ZERO$ 180 190 CONTINUE RD(1) = FT(4,1)*CX - FT(5,1)*CY RD(2) = FT(4,1)*CY + FT(5,1)*CX200 CONTINUE BM(NJ) = FR(3)С DO 280 J = 1; NCT4 IF (JSDN(J) .NE. NJ) GO TO 280 K = 2 + (2 + KU(J) + KV(J)) + KTH(J)IF (K .GT. 7) GO TO 270 M = M + 1JR(M) = JSDN(J)M1 = (K-1) * (K-2) * (K-3)IF (M1 .EQ. 0) GO TO 210 R(M,1) = -FS(NJ,1) + RD(1)GO TO 220 210 R(M,1) = ZERO M2 = (K-1) * (K-4) * (K-5)220 IF (M2 .EQ. 0) GO TO 230 R(M,2) = -FS(NJ,2) + RD(2)GO TO 240 230 $R(M_{2}) = ZERO$ M3 = (K-2) * (K-4) * (K-6)240 IF (M3 .EQ. 0) GD TJ 250 $R(M_{3}) = -FS(N_{3}) + FT(6_{1})$ GO TO 280 250 R(M,3) = ZEROGO TO 280 270 BM(NJ) = ZERO CONT INUE 280 С c>--> PRINT REACTIONS C CALL HEADNG PRINT 2100 DO 300 I = 1, M 300 PRINT 2030, JR(I), (R(I,J), J = 1, 3)RETURN END

```
С
     SUBROUTINE OUTPUT
C
С
C
                         THIS SUBROUTINE CONTROLS OUTPUT
С
                         FOR STATIC AND DYNAMIC SOLUTIONS
C
С
Ĉ
С
       SUBROUTINE OUTPUT (IDPT, TIME, UD, VD)
С
       IMPLICIT REAL # 8(A-H, O-Z)
       COMMON /BMAST/ BMASS(25), BM(25), T(24), SH(24)
COMMON /CONST/ ZERO, P5, DNE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE
       COMMON /XSECT/ B1(25), B2(25), B3(25), CG(25), PC(25)
       DIMENSION UD(1), VD(1)
 2000 FORMAT (//20X, 'MAXIMUM RESPONSE, TIME = ', 1PD12.4//30X,
1 'QUANTITY',8X,'BAR DR',8X,'VALUE'/46X,'STATION'//30X,
                *THRUST*, 11X, I4, 5X, 1PD12.4/30X, *MOMENT*, 11X, I4, 5X,
      2
                1PD12.4/30X, * SHEAR*, 12X, 14, 5X, 1PD12.4/30X, *X DISPL*, 10X,
      3
 4 I4,5X, 1PD12.4/30X,'Y DISPL',10X,I4,5X,1PD12.4)
2010 FORMAT (//20X, 'COMPLETE RESPONSE, TIME = ', 1PD12.4//22X,'STA.',
                5X, * X DISPL', 5X, *Y DISPL', 6X, *MOMENT', 6X, *SHEAR',
      1
                7X, "THRUST", 4X, "CENTROID"/)
      2
 2020 FORMAT (22X, I3, 3X, 1P3D12.4, 24X, D12.4)
 2030 FORMAT (64X, 1P2D12.4)
С
             IOPT = 1, ONLY MAXIMUM VALUES WILL BE PRINTED
IOPT = 2, COMPLETE RESPONSE WILL BE PRINTED
C>-->
          1F
С
          IF
       CALL HEADNG
       GO TO (100, 160), IOPT
          SEARCH FOR MAXIMUM VALUES OF DISPLACEMENTS AND INTERNAL FORCES
C>-->
  100 TMAX = ZERO
       SHMAX = ZERO
       BMMAX = BM(1)
       UMAX = UD(1)
       VMAX = VD(1)
       JB = 1
       JU = 1
       JV = 1
       DO 150 I = 2, NJ
          IF (DABS(T(I-1)) .LT. DABS(TMAX)) GO TO 110
                TMAX = T\{I-1\}
                JT = I - 1
  110
          IF (CABS(SH(I-1)) .LT. DABS(SHMAX)) GO TO 120
                SHMAX = SH(I-1)
                JS = I - 1
          IF (DABS(BM(I)) .LT. DABS(BMMAX)) GO TO 130
  120
                BMMAX = BM(I)
                JB = I
          IF (DABS(UD(I)) .LT. DABS(UMAX)) GO TO 140
  130
                UMAX = UD{I}
                JU = I
          IF (DABS(VD(I)) .LT. DABS(VMAX)) GO TO 150
  140
      .
                VMAX = VD(I)
                JV = I
  150 CONTINUE
          PRINT MAXIMUM VALUES
<--<2
       PRINT 2000, TIME, JT, TMAX, JB, BMAX, JS, SHMAX, JU, UMAX, JV, VMAX
       RETURN
          PRINT COMPLETE RESPONSE
c>-->
  160 PRINT 2010, TIME
       I = 1
       PRINT 2020, I, UD(I), VD(I), BM(I), CG(I)
       DO 170 I = 2, NJ
          PRINT 2030, SH(I-1), T(I-1)
          PRINT 2020, I, UD(I), VD(I), BM(I), CG(I)
  170 CONTINUE
       RETURN
       END
```

C SUBROUTINE DYNAM C C * * * * * * C Ċ THIS SUBROUTINE CONTROLS THE DYNAMIC PROCESS с С С SUBROUTINE DYNAM С IMPLICIT REAL # 8(A-H, O-Z) COMMON /AKCEL/ DDU(25), DDV(25) COMMON /ARCXY/ X(25), Y(25), XLJ(24), SPAN COMMON /BMAST/ BMASS(25), BM(25), T(24), SH(24) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE COMMON /DISPL/ U(25), V(25), TH(25), UD(25), VD(25) COMMON /IGEOM/ PHIO(25), THO(24), IBRK, ISYM COMMON /LOADS/ QX(25), QY(25), QZ(25), QXD(25), QYD(25) COMMON /TABL1/ GRAV, TLIM, KEEP(7), ISTAT, ISOPT, NDL, IDOPT, NOUT, ISELFW COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26), JSDN(10), KU(10), KV(10), KTH(10), NCT4 1 CDMMON /TABL6/ JI6(20), QX16(20), QY16(20), JM6(20), QXM6(20), QYM6(20), JL6(20), QXL6(20), QYL6(20), NCS(20),NPDL COMMON /TIMEF/ F0, TR, TD, FT, TIME, DTIME, TDTIME, IND, INTVL COMMON /XSECT/ B1(25), B2(25), B3(25), CG(25), PC(25) 1 DIMENSION UD0(25), VD0(25), DU0(25), DV0(25), DDU0(25), DDV0(25) DIMENSION ADDU(25), ADDV(25), DU(25), DV(25) DIMENSION JI(20), QXI(20), QYI(20), JM(20), QXM(20), JL(20), QXL(20), QYL(20) 1 С 2020 FORMAT(//20X, 'SOLUTION FOR DYNAMIC LOADING NO. ',13///22X,'STA.', 1 7X,'POINT',8X,'DYNAMIC LOADS',10X,'STATIC LOADS',7X, 'DEPTH OF'/23X, 'NO.', 7X, 'MASS', 5X, 'HORIZONTAL VERTICAL', 3X, 'HORIZONTAL VERTICAL CENTROID'//) 2 3 2030 FORMAT (//20X, 'FAILURE DID NOT OCCJR IN SPECIFIED TIME LIMIT', /20X, 'ELAPSED TIME = ', 1PD12.4) 1 2040 FORMAT (22X, I3, 3X, 1P6D12.4) DATA DM3, DM1, SIX / 1.0D-03, 1.0D-01, 6.0D00 / С ADJUST STATIC CONFIGURATION OF THE STRUCTURE C> Ċ TO CONFORM TO DYNAMIC PROCESS - INITIALIZE VARIABLES С CALL ADJUST(1) C START DYNAMIC SOLUTION C> C NCL = 0DO 400 N = 1, NDL C C>--> ' SET UP DYNAMIC LOADS AT THE JOINTS CALL HEADNG PRINT 2020, N KNOUT = 0 NCI = NCL + 1 NCL = NCL + NCS(N)NC = 0DO 230 I = NCI, NCLNC = NC + 1JI(NC) = JI6(I)JM(NC) = JMG(I)JL(NC) = JL6(I)QXI(NC) = QXI6(I)QYI(NC) = QYI6(I)QXM(NC) = QXM6(I)QYM(NC) = QYM6(I)QXL(NC) = QXL6(I)QYL(NC) = QYL6(I)CONT INUE 230 С CALL DFORCE (NC, JI,QXI,QYI, JM,QXM,QYM, JL,QXL,QYL, QXD,QYD)

SUBROUTINE DYNAM C (CONTINUED) C C>--> INITIALIZE FOR DISPLACEMENTS, VELOCITIES AND ACCELERATIONS AND PRINT OUT POINT MASSES, DYNAMIC LJADS AND DEPTHS OF CENTROIDS C DO 240 I = 1, NJ UDO(I) = ZERO VDO(I) = ZERODUO(I) = ZERO DVO(I) = ZERO DDUO(I) = ZERODDVO(I) = ZEROPRINT 2040, I, BMASS(I), QXD(I), QYD(I), QX(I), QY(I), CG(I) 240 CONT INUE C CHECK FOR TYPE OF DYNAMIC LOADING C>--> IF (LDTYPE .NE. 1) GD TJ 260 С IF DYNAMIC LOADING IS AN IMPULSE, SET INITIAL VELOCITIES C>--> DO 250 I = 1, NJ DUO(I) = QXD(I) / BMASS(I) DVO(I) = QYD(I) / BMASS(I)250 CONT INUE GO TO 280 C IF DYNAMIC LOADING IS A PULSE, SET INITIAL ACCELERATIONS (>-->260 INTVL = 1IF (TR .EQ. ZERO) INTVL = 2 IF (INTVL .NE. 2) GO TO 310 DO 270 I = 1, NJ $DDUO(I) = FO \neq QXD(I) / BMASS(I)$ $DDVO(I) = FO \neq QYD(I) / BMASS(I)$ CONTINUE 270 С REVISE VELOCITIES AND ACCELERATIONS FOR UNVIELDING SUPPORTS C>--> DO 300 I = 1, NCT4 280 IF (KU(I) .NE. 1) GO TO 290 J = JSDN(I)DUO(J) = ZERO DDUO(J) = ZERD 290 IF (KV(I) .NE. 1) GD TD 300 J = JSDN(I)DVO(J) = ZERO DDVO(J) = ZERO300 CONTINUE C C>--> START NUMERICAL ITERATIVE PROCEDURE INCREMENT TIME BY TIME INTERVAL C>--> TIME = TIME + DTIME 310 IF (LDTYPE .NE. 1) CALL FTIME SET INITIAL ITERATION COUNTER C>--> ITER = 0C COMPUTE DYNAMIC DISPL. AND VELOCITIES AND ASSUME ACCELERATIONS C>--> DO 320 I = 1, NJ UD(I) = UDO(I) + DTIME*DUO(I) + P5*DTIME*DTIME*DDUD(I) VD(I) = VDD(I) + DTIME*DVO(I) + P5*DTIME*DTIME*DDVO(I) DU(I) = DUO(I) + DTIME*DDUO(I) DV(I) = DVO(I) + DTIME*DDVO(I) ADDU(I) = DDUO(I)ADDV(I) = DDVO(I)CONTINUE 320 С C>--> COMPUTE CURVATURES, STRAINS, INTERNAL FORCES AND ACCELERATIONS CALL JCURVT CALL FORCE (0) CALL ACCEL С ASSUME CONVERGENCE AND INCREASE ITERATION COUNTER C>--> 330 KONV = 1ITER = ITER + 1

С SUBROUTINE DYNAM (CONTINUED) С DD 340 I = 1, NJ COMPUTE DIFFERENCE BETWEEN ASSUMED AND COMPUTED ACCELERATIONS (>--> DELDD = DDU(I) - ADDU(I)CHECK FOR CONVERGENCE OF HORIZONTAL ACCELERATIONS (>-->COMP = DABS(DM3*DDU(I)) IF (COMP .LT. DM1) COMP = DM1 IF (DABS(DELDD) .GT. COMP) KONV = 0 CALCULATE NEW HORIZONTAL DISPLACEMENTS AND VELOCITIES C>--> UD(I) = UD(I) + DTIME*DTIME*DELDD/SIX DU(I) = DU(I) + P5*DTIME*DELDD C>--> SET NEW ASSUMED HORIZONTAL ACCELERATIONS ADDU(I) = DDU(I)C>--> COMPUTE DIFFERENCE BETWEEN ASSUMED AND CALCULATED ACCELERATIONS DELDD = DDV(I) - ADDV(I)CHECK FOR CONVERGENCE OF VERTICAL ACCELERATIONS $() \rightarrow -)$ COMP = DABS(DM3*DDV(I)) IF (COMP +LT. DM1) COMP = DM1 IF (DABS(DELDD) +GT. COMP) KJNV = 0 CALCULATE NEW VERTICAL DISPLACEMENTS AND VELOCITIES C>--> VD(I) = VD(I) + DTIME*DTIME*DELDD/SIX DV(I) = DV(I) + P5*DTIME*DELDD SET NEW ASSUMED VERTICAL ACCELERATIONS C>--> ADDV(I) = DDV(I)CONTINUE 340 c C>--> COMPUTE CURVATURES, STRAINS, INTERNAL FORCES AND ACCELERATIONS CALL JCURVT CALL FORCE (KONV) CALL ACCEL TEST FOR CONVERGENCE AND MAXIMUM NUMBER OF ITERATIONS C>--> IF (KDNV .EQ. 1) GO TD 350 IF (ITER .LT. 7) GO TD 330 TIME = TIME - DTIME DTIME = P5 * DTIMEGO TO 310 350 KNOUT = KNOUT + 1С C>--> SET INITIAL CONDITIONS FOR NEXT STEP DD 360 I = 1, NJ UDO(I) = UD(I)VDO(I) = VD(I)DUO(I) = DU(I)DVO(I) = DV(I)DDUO(I) = DDU(I)DDVO(I) = DDV(I)360 CONT INUE C C>--> EXAMINE FAILURE CONDITIONS AND TEST FOR FAILURE CALL FAIL (TIME) IF (IFAIL .EQ. 1) GO TO 390 IF (KNOUT .LT. NJUT) GD TO 370 PRINT OUT DYNAMIC RESULTS C>--> CALL OUTPUT (IDOPT, TIME, UD, VD) KNOUT = 0C>--> CHECK FOR TIME LIMIT IF (TIME .GE. TLIM) GO TO 380 370 REVISE TIME-STEP INTERVAL TO INSURE CONVERGENCE AND STABILITY C>--> H = 4 - ITER + ITER/5DTIME = (ONE + DM1*H) * DTIME GO TO 310 PRINT OUT ELAPSED TIME AND DYNAMIC RESULTS C>--> PRINT 2030, TIME 380 CALL OUTPUT (IDOPT, TIME, UD, VD) 390 INITIALIZE VARIABLES FOR NEW DYNAMIC LOADING C>--> IF (N .LT. NDL) CALL ADJUST(2) 400 CONTINUE RETURN

END

```
С
      SUBROUTINE ADJUST
С
С
С
С
         THIS SUBROUTINE ADJUSTS STATIC CONFIGURATION OF THE STRUCTURE
0000000
            IN ORDER TO CONFORM DEFORMED STRUCTURE TO DYNAMIC PROCESS
                CALCULATES INITIAL TIME INTERVAL, IF NOT SUPPLIED
                          ELIMINATES RESIDUAL ACCELERATIONS
                                 INITIALIZES VARIABLES
                                     * * * * * * * *
       SUBROUTINE ADJUST (KEY)
С
       IMPLICIT REAL * 8(A-H, O-Z)
       COMMON /AKCEL/ DDU(25), DDV(25)
       COMMON /ARCXY/ X(25), Y(25), XLD(24), SPAN
COMMON /BMAST/ BMASS(25), BM(25), T(24), SH(24)
       COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE
       COMMON /CURVS/ EPSN(10,5),SIGN(10,5),EPSMUL(5),SIGMUL(5),EPSPR(5)
       COMMON /DISPL/ U(25), V(25), TH(25), UD(25), VD(25)
COMMON /EPPHI/ DPHIJ(25), DPHIJS(25), EPSAB(24), EPSABS(24)
       COMMON /EPSOY/ EPSO(25,32), EPS1P(25,32), EPS1N(25,32), ICRAC(25,32)
       COMMON /IGEOM/ PHIO(25), THO(24), IBRK, ISYM
COMMON /LOADS/ QX(25), QY(25), QZ(25), QXD(25), QYD(25)
COMMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9)
       COMMON /SEGM2/ DI(30), DA(30), MI(30)
       COMMON /STIFF/ EI(25), AE(25)
       COMMON /TABL1/ GRAV,TLIM,KEEP(7),ISTAT,ISOPT,NDL,IDOPT,NOUT,ISELFW
COMMON /TIMEF/ FD, TR, TD, FT, TIME, DTIME, TDTIME, IND, INTVL
       COMMON /XSECT/ B1(25), B2(25), B3(25), CG(25), PC(25)
       DIMENSION DZON(9)
       DATA PI, IYES / 3.1415926535D00, 3HYES /
       IF (KEY .EQ. 2) GO TO 170
       IF (DTIME .GT. ZERO) GO TO 100
         COMPUTE INITIAL TIME-STEP INTERVAL
C>-->
       S = ZERC
       SMASS = ZERO
       SEI = ZERO
       DO 90 I = 1, NB
           S = S + XLO(I)
SMASS = SMASS + BMASS(I)
           SEI = SEI + EI(I)
   90 CONTINUE
       H = NJ
       SEI = (SEI + EI(NJ)) / H
       SMASS = (SMASS+BMASS(NJ)) / S
       H = H - ONE
       S = S / H
       DTIME = S / H * S * DSQRT(SMASS/SEI)
  100 DTIME1 = DTIME
          CHECK IF STATIC SOLUTION WAS REQUIRED
C>-->
          (ISTAT .EQ. IYES) GO TO 120
       1 F
C>-->
           IF NOT, SET DISPLACEMENTS, CURVATURES AND STRAINS EQUAL TO ZERO
       DO 110 I = 1, NB
          U(I) = ZERO
          V(I) = ZERO
          DPHIJS(I) = ZERO
          EPSABS(I) = ZERO
  110 CONTINUE
       U(NJ) = ZERO
       V(NJ) = ZERO
       DPHIJS(NJ) = ZERO
       GD TO 170
           CALCULATE AVERAGE STATIC STRAINS AND CURVATURES
C>-->
  120 DO 130 I = 1, NB
           AEB = P5 * (AE(I)+AE(I+1))
           EPSABS(I) = T(I) / AEB
           DPHIJS(I) = BM(I) / EI(I)
  130 CONTINUE
```

```
(CONTINUED)
     SUBROUTINE ADJUST
С
Ċ
      DPHIJS(NJ) = BM(NJ) / EI(NJ)
         DETERMINE NEW CONFIGURATION OF THE STRUCTURE
C>-->
      DO 140 I = 1, NJ
         X(I) = X(I) + U(I)
         Y(I) = Y(I) + V(I)
         UD(I) = ZERO
         VD(I) = ZERO
         DPHIJ(I) = DPHIJS(I)
  140 CONTINUE
         CALCULATE NEW LENGTHS AND SLOPES OF BARS, CURVATURES AT JOINTS
C>-->
      DO 150 I = 1, NB
         DX = X(I+1) - X(I)
         DY = Y(I+1) - Y(I)
         XLO(I) = DSQRT(DX*DX + DY*DY)
         THO(I) = DARSIN(DY/XLO(I))
         EPSAB(I) = EPSABS(I)
         IF (DX .GE. ZERO) GO TO 150
              THI = PI
              IF (DY .LT. ZERO) THI = -PI
THO(I) = THI - THO(I)
  150 CONTINUE
      DO \ 160 \ I = 2, NB
         PHIO(I) = (THO(I)-THO(I-1))/P5/(XLO(I)+XLO(I-1))
  160 CONTINUE
C>-->
         INITIALIZE VARIABLES
  170
         TIME = ZERO
         DTIME = DTIME1
         IFAIL = 0
         FT = ZERO
         IND = 1
C>-->
         INITIALIZE DYNAMIC LOADS, DEPTHS OF CENTROIDS, PARAMETERS
         OF STRESS-STRAIN CURVES, INDICATORS OF CRACKS
С
         DO 220 I = 1, NJ
               QXD(I) = ZERO
               QYD(I) = ZERO
               CG(I) = PC(I)
               DO 190 J = 1, 32
                    EPSO(I,J) = ZERO
                    ICRAC(I,J) = 0
               CONTINUE
  190
               DO 200 J = 1, 9
                    DZON(J) = D(I,J)
  200
               CONT INUE
               CALL SEG (DZON, B1(I), B2(I), B3(I)).
               DO \ 210 \ J = 1, 30
                    M = MI(J)
                    EPS1N(I,J) = EPSMUL(M) = EPSN(5,M)
                    EPS1P(I,J) = EPSMUL(M) + EPSN(6,M)
  210
               CONT INUE
               EPS1N(I,31) = EPSMUL(NSSC) + EPSN(5,NSSC)
               EPS1P(1,31) = EPSMUL(NSSC) * EPSN(6,NSSC)
               EPS1N(I, 32) = EPS1N(I, 31)
               EPS1P(I,32) = EPS1P(I,31)
  220
         CONT INUE
      IF (KEY .EQ. 2) RETURN
      IF (ISTAT .NE. IYES) RETURN
Č>-->
         ADJUST STATIC FORCES IN OPDER TO ELIMINATE RESIDUAL
C.
         ACCELERATIONS DUE TO THE NEW CONFIGURATION OF THE STRUCTURE
      CALL FORCE (1)
CALL OUTPUT (ISOPT, ZERO, U, V)
      CALL ACCEL
      DO 230 I = 1, NJ
         QX(I) = QX(I) - DDU(I) + BMASS(I)
         QY(I) = QY(I) - DDV(I) + BMASS(I)
  230 CONTINUE
      RETURN
      END
```

110

```
С
     SUBROUTINE JCURVT
С
С
C
               THIS SUBROUTINE CALCULATES CURVATURES AT THE JOINTS
AND ALSO STRAINS IN THE BARS
C
C
С
Ċ
                             * * * * * * * * * * * * * *
                                                                                  *
C
       SUBROUTINE JOURNT
С
       IMPLICIT REAL * 8(A-H, J-Z)
COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN
       COMMON /CONST/ ZERD, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE
       COMMON /DISPL/ J(25), V(25), TT(25), UD(25), VD(25)
       COMMON /EPPHI/ DPHIJ(25), DPHIJS(25), EPSAB(24), EPSABS(24)
      COMMON /IGEOM/ PHID(25), THO(24), IBRK, ISYM
COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26),
      1
                        JSDN(10), KU(10), KV(10), KTH(10), NCT4
       DATA PI / 3.1415926535D 00 /
С
          THIM1 = THO(1)
          XLIM1 = XLO(1)
          DO 100 I = 1, NB
                \begin{array}{l} DXU = X(I+1) - X(I) + UD(I+1) - UD(I) \\ DYV = Y(I+1) - Y(I) + VD(I+1) - VD(I) \end{array}
                XLI = DSQRT(DXU+DXU + DYV+DYV)
                EPSAB(I) = XLI / XLO(I) - ONE + EPSABS(I)
                THI = DARSIN (DYV / XLI)
                IF (DXU .GE. ZERO) GO TO 90
                TH = PI
                IF (DYV .LT. ZERD) TH = -PI
                   THI = TH - THI
   90
                DPHIJ(I) = (THI-THIM1)/P5/(XLIM1+XLI)-PHIO(I)+>PHIJS(I)
                IF (IHINGE(I) .EQ. 1) DPHIJ(I) = ZERD
                XLIM1 = XLI
                THIM1 = THI
  100
          CONTINUE
          XLI = XLO(NB)
          THI = THO(NB)
       DPHIJ(NJ) = (THI-THIM1) / P5 / (XLIM1+XLI) - PHIO(NJ) + DPHIJS(NJ)
       IF (ISYM .EQ. 2) DPHIJ(NJ) = (THI/XLI-THIM1/XLIM1)/P5 + DPHIJS(NJ)
       IF (IHINGE(NJ) .EQ. 1) DPHIJ(NJ) = ZER3
       RETURN
       EN D
С
     SUBROUTINE HEADNG
С
¢
С
С
              THIS SUBROUTINE ESTABLISHES HEADING ON PAGE OF OUTPUT
С
C
C
                             * * * * * *
       SUBROUTINE HEADNG
С
       IMPLICIT REAL # 8(A-H, O-Z)
       COMMON /IDENT/ ID1(40), ID2(19), NPROB
 2000 FORMAT (1H1//20X,15HPRDGRAM DYNARCH/20X,
     1
               51HANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES/20X,
               32HUNDER STATIC AND DYNAMIC LOADING//2(20X,20A4/))
      2
 2010 FORMAT (/20X,8HPROBLEM , A4/24X, 19A4)
С
                PRINT 2000, ID1
PRINT 2010, NPROB, ID2
       RETURN
                                                                                   1
       EN D
```

SUBROUTINE FORCE THIS SUBROUTINE CALCULATES INTERNAL FORCES, IE., THRUSTS AND BENDING MOMENTS, BASED ON STRAINS AND CURVATURES SUBROUTINE FORCE (KONV) IMPLICIT REAL * 8(A-H, O-Z) COMMON /BMAST/ BMASS(25), BM(25), T(24), SH(24) COMMON / CONST/ ZERO, P5, DNE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE COMMON /EPPHI/ DPHIJ(25), DPHIJS(25), EPSAB(24), EPSABS(24) COMMON /EPSOY/ EPSO(25,32), EPS1P(25,32), EPS1N(25,32), ICRAC(25,32) COMMON /SEGM1/ D(25,9), E(5,2), GANMA(5), IS(9), MAT(9) COMMON /STEEL/ AB(25), AT(25), DB(25), DT(25) COMMON /STIFF/ EI(25), 4E(25) COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26), JSDN(10), KU(10), KV(10), KTH(10), NCT4 COMMON /XSECT/ B1(25), B2(25), B3(25), CG(25), PC(25) DIMENSION TPEPSO(32), TPEPS1(32), TNEPS1(32), IPCK(32), TEPSO(32), TEPS1P(32), TEPS1N(32), ICK(32), DZON(9) 1 FORMAT (//20X, 'VALJES OFF STRESS-STRAIN CURVE AT STATION', 14, ', FIBER ', 13) EPSA1 = P5 + EPSAB(1)DO 200 I = 1, NJ TEST IF INELASTIC ACTION IS DESIRED IF (INEL .EQ. 1) GO TO 110 IF NOT, CALCULATE THRUSTS AND BENDING MOMENTS, BASED ON ELASTIC STRAINS AND CURVATURES IF (1 .EQ. NJ) GO TO 100 AEB = P5 * (AE(I)+AE(I+1)) T(1) = AEB + EPSAB(1)BM(I) = EI(I) + DPHIJ(I)GO TO 200 IF (I .EQ. NJ) GO TO 120 ESTABLISH AVERAGE STRAIN AT THE JOINT EPSA2 = P5 + EPSAB(1)EPSA = EPSA1 + EPSA2EPSA1 = EPSA2GO TO 125 EPSA = EPSAB(NB)RECALL DEFORMATION HISTORY OF SEGMENTS OF THE SECTION DO 130 J = 1, 9DZON(J) = D(I,J)CONTINUE DO 140 J = 1, 32TPEPSO(J) = EPSO(I, J)TPEPS1(J) = EPS1P(I,J)TNEPS1(J) = EPS1N(I,J)IPCK(J) = ICRAC(I,J)CONTINUE CALL SUBROUTINE INTERN TO FIND NEW DEPTH OF CENTROID, REDEFINE AXIAL AND FLEXURAL STIFFNESSES, DETERMINE A NEW DISTRIBUTION OF STRESSES AND STRAINS, AND CALCULATE

CALL INTERN (DZON, EPSA, DPHIJ(I), TPEPSO, TPEPS1, TNEPS1, IPCK, CG(I), B1(I), B2(I), B3(I), AT(I), DT(I), AB(I), DB(I), TEPSO, TEPSIP, TEPSIN, ICK, TCG, AE(I), EI(I), BM(I), TJR)

BENDING MOMENT AT JOINT I AND THRUST AT RIGHT OF BAR I-1

```
IF (IFAIL - 1) 170, 150, 160
       PRINT 1, I, JF
150
       STOP
160
       IFAIL = 0
```

С

С C C Č C

c c

Ĉ С

С

С

С

C>-->

C>-->

100

110

120

130

140 C

-->

1

2

C> C C C C

С

С

C>--> 125

C>-->

1

1

111

```
С
     SUBROUTINE FORCE
                                                                    (CONTINUED)
C.
C>-->
          CALCULATE AVERAGE THRUST ACTING ON A BAR
  170
          IF (I .EQ. 1) GO TO 180
               T(I-1) = P5 * (TJL+TJR)
  180
          TJL = TJR
         IF (IHINGE(I) .EQ. 1) BM(I) = ZERO
IF (KONV .EQ. 0) GO TO 200
ESTABLISH NEW DEPTH OF CENTROID AND RECORD NEW PARAMETERS
C>-->
C
          OF STRESS-STRAIN CURVES AND INDICATORS OF CRACKS
               CG(I) = TCG
          DO 190 J = 1, 32
               EPSO(I_{J}) = TEPSO(J)
               EPS1P(I,J) = TPEPS1(J)
               EPS1N(I,J) = TNEPS1(J)
               ICRAC(I,J) = ICK(J)
         CONT INUE
  190
  200 CONTINUE
      RETURN
      EN D
С
     SUBROUTINE MASS
Ċ
С
              * *
                   *
                       * * * * * * * * * *
                                               * *
Ċ
     *
۵
     *
                     THIS SUBROUTINE CALCULATES SELF WEIGHT
С
С
С
                        PER UNIT LENGTH AT GIVEN SECTION
                     * * * * * * * * * * * * * * * * * * *
Ç
      SUBROUTINE MASS (DZON, B1, B2, B3, DB, DT, AB, AT, AGAM)
С
      IMPLICIT REAL * 8(A+H, 0+Z)
      COMMON /CONST/ ZERO, P5, DNE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE
      COMMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9)
COMMON /SEGM2/ DI(30), DA(30), MI(30)
      DIMENSICN DZON(1)
С
(>-->
          DEFINE PROPERTIES OF THE SEGMENTS IN THE CROSS SECTION
      CALL SEG (DZON, B1, B2, B3)
C
          CALCULATE SUM OF SEGMENTAL AREAS MULTIPLIED BY SPECIFIC WEIGHT
() \rightarrow - )
      AGAM = ZERO
      DO 100 J = 1, 30
          MM = MI(J)
  100 AGAM = AGAM + DA(J) * GAMMA(MM)
       IF (AT .EQ. ZERO) GO TO 130
С
          ADD CONTRIBUTION OF TOP REINFORCEMENT, IF ANY
C>-->
      DJ 110 J = 1, 30
          IF (DT .GT. DI(J)) GO TO 110
          MM = MI(J)
          GO TO 120
  110 CONTINUE
  120 A3AM = AGAM + AT*(GAMMA(NSSC)-GAMMA(MM))
  130 IF (AB .EQ. ZERD) RETURN
С
C>-->
          ADD CONTRIBUTION OF BOTTOM REINFORCEMENT, IF ANY
        DO 140 K # 1, 30
          J = 31 - K
          IF (DB .LT. DI(J)) GO TO 140
          MM = MI(J)
          GO TO 150
  140 CONTINUE
  150 AGAM = AGAM + AB*(GAMMA(NSSC)-GAMMA(MM))
      RETURN
      END
```

SUBROUTINE ACCEL С С C C * * * * * * * С THIS SUBROUTINE CALCULATES HORIZONTAL AND VERTICAL Ċ COMPONENTS OF ACCELERATIONS OF POINT MASSES, C BASED ON EQUILIBRIUM OF FORCES ACTING ON JOINTS * c c С SUBROUTINE ACCEL C IMPLICIT REAL * 8(A-H, O-Z) COMMON /AKCEL/ DDU(25), DDV(25) COMMON /ARCXY/ X(25), Y(25), XL0(24), SPAN COMMON /BMAST/ BMASS(25), BM(25), T(24), SH(24) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LDTYPE COMMON /DISPL/ U(25), V(25), TH(25), UD(25), VD(25) COMMON /LOADS/ QX(25), QY(25), QZ(25), QXD(25), QYD(25) COMMON /TABL4/ USN(10), VSN(10), THSN(10), IHINGE(26), JSDN(10), KU(10), KV(10), KTH(10), NCT4 1 COMMON /TIMEF/ FO, TRISE, TD, FT, TIME, DTIME, TDTIME, IND, INTVL С C>--> CALCULATE ACCELERATIONS VL = ZEROTL = ZEROCL = ZEROSL = ZERO DO 100 I = 1, NB DX = X(I+1) - X(I) + UD(I+1) - UD(I) DY = Y(I+1) - Y(I) + VD(I+1) - VD(I)XL = DSQRT(DX*DX + DY*DY)TR = T(I)VR = (BM(I+1)-BM(I)) / XL CR = DX / XLSR = DY / XL $QU = FT \neq QXD(I) + QX(I)$ QV = FT = QYD(I) + QY(I)DDU(I) = (TR*CR - TL*CL + VR*SR + VL*SL + QU) / BMASS(I) DDV(I) = (TR*SR - TL*SL + VL*CL - VR*CR + QV) / BMASS(I) VL = VR TL = TR CL = CRSL = SR**100 CONTINUE** $QU = FT \neq QXD(NJ) + QX(NJ)$ $QV = FT \neq QYD(NJ) + QY(NJ)$ DDU(NJ) = (-TL*CL - VL*SL + QU) / BMASS(NJ)DDV(NJ) = (-TL*SL + VL*CL + QV) / BMASS(NJ)С C>--> REVISE VALUES OF ACCELERATIONS FOR UNVIELDING SUPPORTS C DO 120 I = 1, NCT4 IF (KU(I) .NE. 1) GO TO 110 J = JSDN(I)DDU(J) = ZEROIF (KV(I) .NE. 1) GO TO 120 110 J = JSDN(I)DDV(J) = ZERO120 CONTINUE RETURN END

C SUBROUTINE INTERN С С C C THIS SUBROUTINE DETERMINES NEW DEPTH OF CENTROID. C REDEFINES AXIAL AND FLEXURAL STIFFNESSES, с с DETERMINES A NEW DISTRIBUTION OF STRESSES AND STRAINS, AND CALCULATES BENDING MOMENT AND THRUST AT SECTION С C С SUBROUTINE INTERN (DZON, EPSA, PHI, PEPSO, PPEPSI, PNEPSI, IPCK, 1 PCI, B1, B2, B3, AT, DT, AB, DB, EPSO, EPS1P, EPSIN, ICK, DBAR, SAE, SEI, BM, T) 2 С IMPLICIT REAL * 8(A-H, 3-Z) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL,LOTYPE COMMON /SEGM2/ DI(30), DA(30), MI(30) DIMENSION DZON(1), PEPSD(1), PPEPS1(1), PNEPS1(1), IPCK(1), ICK(1), EPSO(1), EPSIP(1), EPSIN(1) 1 DATA DM2 / 1.0D-02 / DETERMINE PROPERTIES OF THE SEGMENTS OF THE SECTION C>--> CALL SEG (DZON, 81, 82, 83) C INITIALIZE ITERATION COUNTER C>--> ITER = 0С C>--> ASSUME DEPTH OF CENTROID AND INITIALIZE VARIABLES XBAR = PCI100 DBAR = XBARXBAR = ZEROSAE = ZERO SEI = ZERO BM = ZEROT = ZERO MB = 0MT = 0 С DETERMINE NEW DISTRIBUTION OF STRAINS AND STRESSES, C> С AND FIND MODULI OF ELASTICITY С $D0 \ 120 \ J = 1, 30$ CALCULATE DISTANCE OF SEGMENT TO THE CENTROID OF THE SECTION C>--> DJ = DI(J) - DBARCALCULATE AVERAGE STRAIN IN THE SEGMENT C>--> EPSJ = EPSA + PHI * DJC C>--> FIND AVERAGE STRESS ACTING ON THE SEGMENT AND DETERMINE MODULUS OF ELASTICITY C CALL SEARCH (MI(J), PEPSO(J), PPEPS1(J), PNEPS1(J), IPCK(J), EPSJ, EPSJ(J), EPS1P(J), EPS1N(J), ICK(J), EJ, SIGMA) 1 С c>--> TEST IF SEARCH WAS SUCCESSFUL IF (IFAIL .EQ. 1) GD TJ 170 ADD CONTRIBUTION OF THE SEGMENT C>--> DAE = DA(J) * EJSAE = SAE + DAEXBAR = XBAR + DAE + DI(J)SEI = SEI + DAE*DJ*DJ DF = DA(J) + SIGMAT = T + 0F BM = BM + DF * DJIF (MT .NE. 0) GO TJ 113 TEST FOR TOP STEEL IN THE SEGMENT C>--> IF (DT .GT. DI(J)) GO TO 110 MT = MI(J)EMT = EJ SIGT = SIGMA 110 IF (MB .NE. 0) GO TO 120

114

```
SUBROUTINE INTERN
С
                                                                 (CONTINUED)
C
C>-->
         TEST FOR BOTTOM STEEL IN THE SEGMENT
         IF (CB .GT. DI(J)) GD TJ 120
               MB = MI(J)
               EMB = EJ
               SIGB = SIGMA
  120 CONTINUE
С
C>
  -->
         CHECK PRESENCE OF TOP REINFORCEMENT
      J = 31
$
      IF (AT .NE. ZERO) GD TO 130
         EPSO(J) = PEPSO(J)
         EPS1P(J) = PPEPS1(J)
         EPS1N(J) = PNEPS1(J)
         ICK(J) = IPCK(J)
         GO TO 140
C
C>-->
         ADD CONTRIBUTION OF TOP REINFORCEMENT
  130 \text{ DJ} = \text{DT} - \text{DBAR}
      EPSJ = EPSA + PHI * DJ
      CALL SEARCH (NSSC, PEPSO(J), PPEPS1(J), PNEPS1(J), IPCK(J), EPSJ,
                    EPSO(J), EPSIP(J), EPSIN(J), ICK(J), EJ, SIGMA)
     1
      IF (IFAIL .EQ. 1) GO TO 170
      DAE = AT * (EJ-EMT)
      SAE = SAE + DAE
      XBAR = XBAR + DAE*DT
      SEI = SEI + DAE*DJ*DJ
      DF = AT * (SIGMA-SIGT)
      T = T + DF
      BM = BM + DF \neq DJ
С
C>-->
         CHECK PRESENCE OF BOTTOM REINFORCEMENT
  140 J = 32
      IF (AB .NE. ZERO) GO TO 150
          EPSO(J) = PEPSO(J)
         EPS1P(J) = PPEPS1(J)
          EPSIN(J) = PNEPS1(J)
         ICK(J) = IPCK(J)
         GO TO 160
С
<>-->
         ADD CONTRIBUTION OF BOTTOM REINFORCEVENT
  150 DJ = DB - DBAR
      EPSJ = EPSA + PHI * DJ
      CALL SEARCH (NSSC, PEPSO(J), PPEPS1(J), PNEPS1(J), IPC<(J), EPSJ,
                    EPSD(J), EPS1P(J), EPS1N(J), ICK(J), EJ, SIGMA)
     1
      IF (IFAIL .EQ. 1) GO TO 170
      DAE = AB * (EJ-EMB)
      SAE = SAE + DAE
      XBAR = XBAR + DAE + DB
      SEI = SEI + DAE * DJ * DJ
      DF = AB + (SIGMA - SIGB)
      T = T + DF
      BM = BM + DF * DJ
С
C>-->
          CALCULATE NEW DEPTH OF CENTROID OF THE TRANSFORMED AREA
  160 XBAR = XBAR / SAE
C
  .
          COMPARE ASSUMED AND CALCILATED DEPTHS OF CENTROID
C>-->
          AND RETURN IF AGREEMENT IS ACCEPTABLE
С
      IF (DABS(DBAR-XBAR) .LT. DM2) RETURN
IF NO AGREEMENT, INCREASE ITERATION COUNTER AND START ALL OVER
C>-->
         AGAIN, PROVIDED THAT THE ITERATION LIMIT HAS NOT BEEN EXCEEDED
С
      ITER = ITER + 1
      IF (ITER .LT. 10) GO TO 100
          IFAIL = 2
          RETURN
  170 JF = J
      RETURN
      END
```

```
С
     SUBROUTINE FTIME
С
C
C
Ċ
                 THIS SUBROUTINE DETERMINES FUNCTION OF TIME FT
                     IN CASE OF A TRIANGULAR FORCING PJLSE
     *
C
C
C
      SUBROUTINE FTIME
C
      IMPLICIT REAL # 8(A-H, O-Z)
      COMMON /TIMEF/ FO, TR, TD, FT, TIME, DTIME, TDTIME, IND, INTVL
      DATA ZERO / 0.0D00 /
C
                           ELAPSED TIME IS LESS THAN TIME OF RISING
C>-->
         FOR
               INTVL = 1,
                           ELAPSED TIME IS GREATER THAN TIME OF RISING,
С
         FOR
              INTVL = 2,
С
                           BUT LESS THAN TIME OF DURATION
C
         FOR
                           ELAPSED TIME IS GREATER THAN TIME OF DURATION,
               INTVL = 3_7
C
                            INDICATING RESPONSE AFTER THE FORCING PULSE
С
         LOCATE REGION OF PULSE DIAGRAM AT TIME
c>-->
      GO TO (100, 110, 130), INTVL
  100 IF (TIME .LT. TR) GO TO 150
С
         REDEFINE REGION OF FORCING PULSE DIAGRAM FOR NEXT STEP
C>-->
            INTVL = 2
C
         KEEP INTERVAL OF TIME FOR NEXT STEP AND CALCULATE
C>
  -->
               INTERVAL OF TIME FOR PRESENT STEP
C.
            IND = 0
            TDTIME = DTIME
            DTIME = TR - TIME + DTIME
C
         REDEFINE ELAPSED TIME AND FUNCTION FT
C>-->
            TIME = TR
            FT = FO
            RETURN
  110 IF (TIME .LT. TR) GO TO 145
IF (IND .EQ. 1) GO TO 120
С
() \rightarrow - \rangle
         REDEFINE ELAPSED TIME AND RESTORE INTERVAL OF TIME
            TIME = TIME - DTIME +TDTIME
            DTIME = TDTIME
            IND = 1
  120 IF (TIME .LT. TD) GD TD 160
         REDEFINE REGION OF FORCING PULSE DIAGRAM FOR NEXT STEP
C>-->
            INTVL = 3
            IND = 0
            TDTIME = DTIME
            DTIME = TD - TIME + DTIME
            TIME = TD
            FT = ZERO
            RETURN
  130 IF (TIME .LT. TD) GD TO 155
      IF (IND .EQ. 1) GO TO 140
            TIME = TIME - DTIME + TDTIME
            DTIME = TDTIME
            IND = 1
  140 FT = ZERO
      RETURN
  145 INTVL = 1
      IND = 1
  150 FT = FO * TIME / TR
      RETURN
  155 INTVL = 2
      IND = 1
  160 FT = FO * (TD-TIME) / (TD-TR)
      RETURN
      END
```

С SUBROUTINE FAIL С C č С THIS SUBROUTINE CHECKS FOR COLLAPSE OF THE STRUCTURE, * C BASED ON MAXIMUM DISPLACEMENTS, MAXIMUM SHEAR, * С THRUST-MOMENT INTERACTION AND CRUSHING OF TOP c c AND/OR BOTTOM FIBERS OF THE SECTION С * * * * * * * * * * * * * Ċ SUBROUTINE FAIL (TIME) С IMPLICIT REAL # 8(A-H, D-Z) COMMON /ARCXY/ X(25), Y(25), XLO(24), SPAN COMMON / BMAST/ BMASS(25), BM(25), T(24), SH(24) COMMON /CONST/ ZERO, P5, ONE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE COMMON /DISPL/ U(25), V(25), TH(25), UD(25), VD(25) COMMON /EPPHI/ DPHIJ(25), DPHIJS(25), EPSAB(24), EPSABS(24) COMMON /FAIL1/ SMAX(25), SMAXN(10), JS7N(10), UMAX, VMAX, NST7 COMMON /FAIL2/ BMUL(25), PMUL(25), BMULN(10), PMULN(10), PIAN(9), BIAN(9), EPSU(5), JIA7(10), NIA7 1 CONMON /SEGM1/ D(25,9), E(5,2), GAMMA(5), IS(9), MAT(9) COMMON /XSECT/ B1(25), B2(25), B3(25), CG(25), PC(25) С 2010 FORMAT (//20X, 'FAILURE DUE TO HORIZONTAL DISPL. AT STATION', 14) 2020 FORMAT (//20X, FAILURE DUE TO VERTICAL DEFLECTION AT STATION , 14) 2030 FORMAT (//20X, 'FAILURE DUE TO SHEAR AT BAR', 14) 2040 FORMAT (//20X, FAILURE DUE TO THRUST-MOMENT INTERACTION, * AT STATION*, I4) 1 2050 FORMAT (//20X, FAILURE OCCURRED AT TIME = ', 1PD12.4, ' SECS.'} 2060 FORMAT (//20X, 'FAILURE DUE TO CRUSHING OF TOP FIBERS AT BAR', 14) 2070 FORMAT (/20X, 'FAILURE DJE TO CRUSHING OF BOTTOM FIBERS AT BAR', 14) 2080 FORMAT (//20X, 'FAILURE DUE TO CRUSHING OF TOP FIBERS AT JOINT', 14) 2090 FDRMAT (/20X, 'FAIL. DUE TO CRUSHING OF BOTTOM FIBERS AT JOINT', 14) C C>--> CHECK FOR FAILURE DUE TO HORIZONTAL DISPLACEMENT KEY = 0DD 120 I = 1, NJ TOTAL = UD(I) + U(I)IF (DABS(TOTAL) .LT. UMAX) GO TO 110 IFAIL = 1IF (KEY .EQ. 0) CALL HEADNG KEY = 1 PRINT 2010, I C>--> CHECK FOR FAILURE DUE TO VERTICAL DEFLECTION TOTAL = VD(I) + V(I)110 IF (DABS(TOTAL) .LT. VMAX) GO TO 120 IFAIL = 1IF (KEY .EQ. 0) CALL HEADNG KEY = 1PRINT 2020, I 120 CONTINUE c>--> CHECK FOR FAILURE DUE TO MAXIMUM SHEAR DO 130 I = 1, NB DX = X(I+1) - X(I) + UD(I+1) - UD(I)DY = Y(I+1) - Y(I) + VD(I+1) - VD(I) $XL = DSQRT (DX \neq DY \neq DY)$ SH(I) = (BM(I+1)-BM(I)) / XLIF (DABS(SH(I)) .LT. SMAX(I)) GO TO 130 IFAIL = 1IF (KEY .EQ. 0) CALL HEADNG KEY = 1PRINT 2030, I **130 CONTINUE** CHECK FOR FAILURE DUE TO THRUST-MOMENT INTERACTION C>--> TBIM1 = T(1)DO 200 I = 1, NJUD(I) = UD(I) + J(I)VD(I) = VD(I) + V(I)÷.,

```
С
     SUBROUTINE FAIL
                                                                 (CONTINUED)
С
         IF (I .EQ. NJ) GO TO 135
         TBI = T(I)
         GO TO 138
  135
         TBI = T(NB)
         TJI = P5 * (TBIM1 + TBI)
  138
         TBIM1 = TBI
         BMPC = BN(I) + TJI + (CG(I)-PC(I))
         IF (TJI .GT. ZERD) TJI = ZERD
         TJI = DABS(TJI) / PMUL(I)
         IF (BMPC .GT. ZERO) GO TO 160
         DO 140 J = 2, 5
               IF (TJI-PIAN(J)) 150, 150, 140
  140
         CONT INUE
         GO TO 190
  150
         BMMIN = BMUL(I) * (BIAN(J-1) + (TJI-PIAN(J-1)) /
                             (PIAN(J)-PIAN(J-1)) * (BIAN(J)-BIAN(J-1)))
     1
         IF (BMPC-BMMIN) 190, 190, 200
         DO 170 J = 2, 5
  160
               K = 10 - J
               IF (TJI-PIAN(K)) 180, 180, 170
         CONTINUE
  170
         GO TO 190
         BMMAX = BMUL(I) + (BIAN(K+1) + (TJI-PIAN(K+1)) /
  180
                             (PIAN(K)-PIAN(K+1)) * (BIAN(K)-BIAN(K+1)))
     1
         IF (BMPC .LT. BMMAX) GD TD 200
  190
               IFAIL = 1
               IF (KEY .EQ. 0) CALL HEADNG
               KEY = 1
               PRINT 2040, I
  200 CONTINUE
С
         CHECK FOR FAILURE DUE TO CRUSHING OF EXTREME FIBERS
C>-->
      EPSJ1 = P5 \neq EPSAB(1)
      DO 260'I = 1, NJ
         IF (I .EQ. NJ) GO TO 230
         PHI = P5 * (DPHIJ(I) + DPHIJ(I+1))
         DZONT = P5 * P5 * (D(I+1)+D(I+1+1))
DZONB = P5 * P5 * (D(I+8)+D(I+1+8)+D(I+1+9))
         TCG = P5 * (CG(I) + CG(I+1))
         EPST = EPSAB(I) + PHI*(DZONT-TCG)
         EPSB = EPSAB(I) + PHI*(DZONB-TCG)
         CHECK FOR FAILURE DUE TO CRUSHING DF TOP FIBERS AT BAR
C>-->
         IF (EPST .GE. ZERO) 30 TO 210
EPST = - EPST
               J = MAT(1)
               IF (EPST .LT. EPSU(J)) GO TO 210
                  IFAIL = 1
                  IF (KEY .EQ. 0) CALL HEADNG
                  KEY = 1
                  PRINT 2060, I
         CHECK FOR FAILURE DUE TO CRUSHING OF BOTTOM FIBERS AT BAR
C>-->
  210
         IF (EPSB .GE. ZERO) 30 TO 220
               EPSB = - EPSB
               J = MAT(9)
               IF (EPSB .LT. EPSU(J)) GO TO 220
                  IFAIL = 1
                  IF (KEY .EQ. 0) CALL HEADNG
                  KEY = 1
                  PRINT 2070, I
         EPSJ2 = P5 + EPSAB(I)
  220
         EPSJ = EPSJ1 + EPSJ2
         EPSJ1 = EPSJ2
         GO TO 240
  230
         EPSJ = EPSAB(NB)
  240 DZONT = P5 * D(I_{1})
      DZONB = P5 * (D(I_{1}8) + D(I_{1}9))
      EPST = EPSJ + DPHIJ(I)*(DZONT-CG(I))
      EPSB = EPSJ + DPHIJ(I)*(DZONB-CG(I))
```

```
С
     SUBROUTINE FAIL
                                                               (CONTINUED)
С
C>-->
         CHECK FOR FAILURE DUE TO CRUSHING OF TOP FIBERS AT JOINT
         IF (EPST .GE. ZERO) GO TO 250
              EPST = - EPST
              J = MAT(1)
              IF (EPST .LT. EPSU(J)) GO TO 250
                 IFAIL = 1
                 IF (KEY .EQ. 0) CALL HEADNG
                 KEY = 1
                 PRINT 2080, I
C>-->
         CHECK FOR FAILURE DUE TO CRUSHING OF BOTTOM FIBERS AT JOINT
  250
         IF (EPSB .GE. ZERO) GO TO 260
              EPSB = - EPSB
               J = MAT(9)
              IF (EPSB .LT. EPSU(J)) GO TO 250
IFAIL = 1
                 IF (KEY .EQ. 0) CALL HEADNG
                 KEY = 1
                 PRINT 2090, I
  260 CONTINUE
      IF FAILURE HAVE OCCURRED, PRINT TIME OF FAILURE
IF (IFAIL .EQ. 1) PRINT 2050, TIME
C>-->
      RETURN
      EN D
     SUBROUTINE SEARCH
С
C
С
Ċ
C
           THIS SUBROUTINE SEARCHES FOR AVERAGE STRESS AND MODULUS
     *
                 OF ELASTICITY IN THE SEGMENT OF THE SECTION,
C
C
     *
               BY CONSULTING THE STRESS-STRAIN CURVE AND TAKING
     *
Č
C
               INTO ACCOUNT THE STRAIN HISTORY OF THE SEGMENT;
                 DEFINES TEMPORARY INDICATOR OF CRACKING AND
С
С
С
                     PARAMETERS OF THE STRESS-STRAIN CURVE
             С
      SUBROUTINE SEARCH (MAT, PEPSO, PPEPS1, PNEPS1, IPCK, EPSJ,
                         EPSO, EPS1P, EPS1N, ICK, E, SIGMA)
     1
С
      IMPLICIT REAL # 8(A-H, 0-Z)
      COMMON /CONST/ ZERO, P5, DNE, NB, NJ, NSSC, IFAIL, JF, INEL, LDTYPE
      COMMON /CURVS/ EPSN(10,5),SIGN(10,5),EPSMUL(5),SIGMUL(5),EPSPR(5)
      DIMENSION EPS(5), SIG(5)
C
      EPSO = PEPSO
      EPS1P = PPEPS1
      EP$1N = PNEP$1
      ICK = IPCK
С
         CALCULATE INITIAL TANGENT MODULUS IN COMPRESSION
<>-->
      FAC = SIGMUL(MAT) / EPSMUL(MAT)
      ECI = FAC + SIGN(5,MAT) / EPSN(5,MAT)
      ENTER = EPSJ - PEPSO
         CHECK FOR TENSION OR COMPRESSION
C>-->
      IF (ENTER .LT. ZERO) GD TO 170
C>-->
         CHECK FOR PREVIOUS CRACKING
      IF (IPCK .EQ. 1) GO TO 110
      IF (ENTER .GT. EPSPR(MAT))
                                  GO TO 100
        DEFINE MODULUS OF ELASTICITY AND STRESS FOR JNCRACKED SECTION
C>-->
      E = ECI
      SIGMA = ECI + ENTER
      RETURN
  100 \text{ ICK} = 1
```

```
SUBROUTINE SEARCH
                                                                (CONTINUED)
C
c>-->
         CALCULATE INITIAL TANGENT MODULUS IN TENSION
 110 ETI = FAC * SIGN(6, MAT) / EPSN(6, MAT)
         GENERATE ACTUAL STRESS-STRAIN CURVE, TENSION SIDE
C>-->
      TEPS = PPEPS1 / EPSMUL(MAT)
      DO 120 J = 1, 4
         K = J + 6
        · IF (TEPS .LT. EPSN(K, MAT)) GO TO 130
  120 CONTINUE
      GO TO 230
  130 E< = FAC * (SIGN(K,MAT)-SIGN(K-1,MAT))/(EPSN(K,MAT)-EPSN(K-1,MAT))
      SIG(1) = SIGMUL(MAT)*SIGN(K-1,MAT) +
               EK * (PPEPS1-EPSN(K-1,MAT)*EPSMUL(MAT))
     1
      EPS(1) = SIG(1) / ETI
      IF (ENTER .GT. EPS(1)) GD TJ 140
         DEFINE AVERAGE STRESS AND INITIAL TANGENT MODULUS OF ELASTICITY
C>-->
      SIGMA = ETI + ENTER
      E = ETI
      RETURN
  140 D3 150 J = K, 10
         L = J - K + 2
         SIG(L) = SIGN(J,MAT) * SIGMUL(MAT)
         EPS(L) = EPS(1) + EPSN(J, MAT) * EPSMUL(MAT) - PPEPS1
         IF (ENTER .LT. EPS(L)) GJ TO 160
  150 CONTINUE
c>-->
         WARNING: VALUES OFF STRESS-STRAIN CURVE
      GO TO 230
  160 EK = (SIG(L)-SIG(L-1)) / (EPS(L)-EPS(L-1)).
         DEFINE AVERAGE STRESS AND SECANT MOD. OF ELASTICITY IN TENSION
C>-->
      SIGMA = SIG(L-1) + EK + (ENTER-EPS(L-1))
      EPS1P = PPEPS1 + ENTER - EPS(1)
      E = SIGMA / EPS1P
      EPSO = PEPSO + ENTER - SIGMA/ETI
      RETURN
c>-->
         GENERATE ACTUAL STRESS-STRAIN CURVE, COMPRESSION SIDE
  170 TEPS = PNEPS1 / EPSMUL(MAT)
      DO 180 J = 1, 4
         K = 5 - J
         IF (TEPS .GT. EPSN(K, MAT)) GO TO 190
  180 CONTINUE
         WARNING: VALUES OFF STRESS-STRAIN CURVE
C>-->
      GO TO 230
  190 EK = FAC * (SIGN(K,MAT)-SIGN(K+1,MAT))/(EPSN(K,MAT)-EPSN(K+1,MAT))
      SIG(1) = SIGMUL(MAT) * SIGN(K+1,MAT) +
     1
               EK * (PNEPS1 - EPSN(K+1,MAT)*EPSMUL(MAT))
      EPS(1) = SIG(1) / ECI
      IF (ENTER .LT. EPS(1)) 30 TO 200
         DEFINE AVERAGE STRESS AND INITIAL TANGENT MODULUS OF ELASTICITY
(> - - >
      SIGMA = ECI * ENTER
      E = ECI
      RETURN
  200 DO 210 J = 1, K
         L = K - J + 1
         SIG(J+1) = SIGN(L,MAT) * SIGMUL(MAT)
         EPS(J+1) = EPS(1) + EPSN(L,MAT)*EPSMJL(MAT) - PNEPS1
         IF (ENTER .GT. EPS(J+1)) GD TO 220
  210 CONT INUE
         WARNING: VALUES OFF STRESS-STRAIN CJRVE
C>-->
      GD TO 230
 220 EK = (SIG(J+1)-SIG(J)) / (EPS(J+1)-EPS(J))
--> DEFINE AVERAGE STRESS AND SECANT MOD. OF ELAST. IN COMPRESSION
C>-->
      SIGMA = SIG(J) + EK * (ENTER-EPS(J))
      EPS1N = PNEPS1 + ENTER - EPS(1)
      E = SIGMA / EPSIN
      EPSO = PEPSO + ENTER - SIGMA/ECI
      RETURN
  230 IFAIL = 1
      RETURN
      END
```

APPENDIX C APPENDIX C APPENDIX C

PROGRAM DYNARCHROGRAM DEYNARCHBAGARAWNEDNAACHDATAGUNNEUTFOR DATA INPUT

IDENTIFICATION OF RUN

Two alphameric cards

·	(20A4)
l	80
· · · · · · · · · · · · · · · · · · ·	(20A4)
1	80

IDENTIFICATION OF PROBLEM

One alphameric card each problem Program terminates execution, if NPROB (Problem Name) is blank

NPROB	Description of Problem	(20A4)
14	10	80

TABLE 1--PROBLEM CONTROL DATA

Two cards each problem First card

2	2	3	<u>}</u>		<u>+</u> _1	5	5	6	5		7	i Ye	а 	b	с	<u>d</u>	e	f	g	
 7	1	1 2	1 5	1 7	2	2	2 5	2 7	3 0	3 2	3 5	3	4 0	45	50	55	60	65	70	
W	he	re:																		
2	t	o 7	. –	En TA	ten BLE	ES :	KEE 2 t	P" 0	to 7) r	eta	in	da	ta fr	om pr	reviou	s pr	oblem	for	
a) ISTAT (A3) - Option for static solution Enter "YES" or "NO", cols. 38-40																				
b)	ISE	LFV	N (15)) –	0p = =	ti 0, 1,	on Se Se	to elf elf	CO -we -we	nsi igh	ide nț nt	r sel will will	f-wei not b be ca	ght e tak lcula	en i ted	nto a inter	ccount nally	

- and added to static loads

TABLE 1 (Continued)

d) NAI	DL ((15)	- Numb ber reta	er of of add ined f	dynamic itional rom pre	loadi dynam vious	ings f nic lo probl	or the adings em. M	prob , if aximu	lem or TABLE m: 20	num- 6 is
e) LD	TYPE ((15)	- Opti = 0, = 1,	on for Forci Impul	type o ng puls se	f dyna e	amic 1	oading			
f) ID(OPT ((15)	- Dyna = 1, = 2,	mic re Only Compl	sults o maximum ete res	utput value ponse	optio es pri	n nted			
g) NOU) TL	(15)	- Outp Ente time	ut int r l if inter	erval f output val	or dyr is de	namic esired	soluti at th	on e end	ofev	ery
Secon	d Caro	1									
 INEL 2 6 10	IBRK 1 15	15YM 20	NB 25	31	SPAN 40	GRAV	/ 50	TLIM	60	DTIME 7	 0
Where	:										
INEL	(15)	-	Optic = 0, = 1,	n for Elasti Inelas	range o c respo tic res	f resp nse or ponse	oonse nly desir	ed			
IBRK	(15)	-	Optic = 0, = 1,	n for Axis d Broken frame. Coordi next c	geometr efined line s A dum nates o ards	y of s by a s tructu my fur f all	struct suppli ire, f nction joint	ure ed fun or exa FX mu s must	ction mple, st be be s	FX a por used. upplie	tal d on
ISYM	(15)	-	Defir = 0, = 1, = 2,	ition Nonsym Struct Symmet SPAN = Symmet Only h SPAN = Last s	of symm metric ure sym ric or total ric str alf str Half s tation	etry struct metric nonsym span c ucture ucture pan of at the	ture abou mmetri of the e and e is s f the e crow	t a ve c load struc load olved struct n of t	rtica ture ure he ar	1 axis ch	
NB	(15)	-	Numbe Maxim	r of b uum: 24	ars in (For p	the mo rogram	odel s n as n	tructu ow wri	re tten)		

SPAN (E10.3) - Span of the structure

- GRAV (E10.3) Acceleration of gravity, necessary for calculation of concentrated point masses. Compatible units must be used
- TLIM (E10.3) Time limit for the dynamic solution Unit of time is always the second
- DTIME (E10.3) Initial interval of time for the dynamic solution If left blank, an approximate tentative value will be internally calculated

If IBRK is left blank, the station coordinates are calculated internally, by using supplied function FX. The next set of cards are needed only if IBRK is set equal to 1, and the joint coordinates are read in ten values per card (10F8.0):

TABLE 2--CROSS SECTION DESCRIPTION

Minimum of nine cards per problem No card, if TABLE 2 is retained from previous problem

A) Control Card

Minimum of two cards: one for the first station and one for the last station. Maximum of ten cards.



Where:

JS (I5) - Station number: enter "1" on the first card and the number of last station (NB + 1) on the last card

- B1 (E10.3) Width of top flange at station JS. See Figure 3.2
- B2 (E10.3) Thickness of the web at station JS
- B3 (E10.3) Width of bottom flange at station JS

IEND (A3) - Enter "END" if JS is the number of last station

÷---

Three cards for each control card. See Figure 3.2



Remark: Depths may change from station to station, but segment numbers and material identification <u>may not</u>. For this reason, IS(K) and MAT(K) may be entered only for the last station.

C) Reinforcement Description

Minimum of one card; maximum of ten cards for all problems.

JRN		DTN	ATN	DBN	ABN	
6 10	21	30	40	50	60	

Where:

JRN (I5) - Number of station where a change in reinforcement occurs. For uniform distribution of reinforcement all over the structure, set JRN equal to the last station number and enter only one card. If there is no reinforcement, leave columns 21 to 60 blank

- DTN (E10.3) Depth of centroid of top reinforcement from top
- ATN (E10.3) cross section area of top reinforcement
- DBN (E10.3) Depth of centroid of bottom reinforcement from top
- ABN (E10.3) Cross section area of bottom reinforcement

Remarks: All cards in TABLE 2 must be in ascending order of station numbers, starting with station 1;

Values for omitted stations will be linearly interpolated between input stations;

Omitted segments are assumed to be equally spaced within the zone.

TABLE 3--STRESS-STRAIN CURVES

Minimum of five cards; maximum of 21 cards No card, if TABLE 3 is retained from previous problem Specification according to Figure 3.3

A) Control Card

Only one card required

NSSC	
6 10	

Where:

NSSC (I5) - Number of stress-strain curves to be input. Maximum of five curves allowed. Curves must be input according to material identification in TABLE 2. Last curve input is used for all reinforcement, if any

B) Specific Weight and Stress Values

Two cards for each curve



SIGN(J) (F8.0) - Factor to be internally multiplied by the stress multiplier, in order to obtain the stress at the jth point of the stress-strain curve. Input must proceed from most negative to most positive value. Ten values must be supplied

C) Strain Values

Two cards for each curve

			,	EPSMUL	EPSU	EPSPR				
			2	30	40)	50		· · · · · · · · · · · · · · · · · · ·	
EPSN	(1) EPSN 8	1(2) 16	24	E	PSN(J), 40	J=1,10 48	56	64	(10F	8.0)
	Where:									
	EPSMUL	(E10.3)	-	Strain m	ultiplie	er; may	not be	e zero or	blank	
	EPSU	(E10.3)	-	Ultimate	strain	in com	pressio	on for the	e materia	1]
	EPSPR	(E10.3)	-	Ultimate sistance sidered, l.O E-8.	strain of conc specify Leave	in ten crete i ⁄a ver blank	sion fo n tens y smal for oth	or concre ion is no l value, ner mater	te. If r t to be c for insta ials	re- con- ince,
	EPSN(J)	(F8.0)	-	Factor t multipli jth poin proceed Ten valu	o be int er, in c t of the from mos es must	cernall order t stres st nega be sup	y mult o obta s-stra tive to plied	iplied by in the str in curve. o most pos	the stra rain at t Input m sitive va	the Sust Uust
	Remarks	s: At le	ast	: one str	ess-stra	ain cur	ve must	t be avai	lable;	

Program terminates execution if any strain limit is exceeded, i.e., if points falloff stress-strain curve.

TABLE 4--SPECIFIED CONDITIONS

Minimum of one card each problem. For stability of structure, at least three displacements must be specified. No card, if TABLE 4 is retained from previous problem.



Where:

TABLE 4 (Continued)

JSDN	(15)	-	Station number where a displacement is to be speci- fied
USN	(E10.3)	-	Value of specified horizontal displacement. May be left blank if displacement is zero
VSN	(E10.3)	-	Value of specified vertical displacement
THSN	(E10.3)	-	Value of specified rotation
IEND	(A3)	-	Enter "END" if last card in TABLE 4
Code		-	Enter "1" in column 18 to specify horizontal displace- ment, "1" in column 19 or 20 to specify vertical dis- placement or rotation. Then 001 will specify only a rotation.
			A hinge may be specified at joint JSDN, by entering

a code number greater than 111. Generally 999 is used.

If ISYM is set equal to 2 in TABLE 1, Code must be 101 for the last station

TABLE 5--STATIC LOADS

Minimum of one card; maximum of twenty cards per run. No card, if static solution is not required, i.e., if ISTAT in TABLE 1 is set different from "YES".

Data in TABLE 5 are cumulative. If TABLE 5 is retained, present specification is added to previous table; if no additional load is added to previous table, a blank card is used with "END" in columns 78-80.

J	115	QXI	5	QYI5	, "	JM5	QXM	5	QYM5		JL5	QXL5		QYL5		IEND
															L	
Т	5	5	15	· · · · ·	25	30		40		50	55		65	7	5	78-80
		Where	e:		. 4	<i>u. 1</i>										
		J15		(15)	- Ņ	umber	of	stat	ion							
		QXI5	(E1	0.3)	– M	agnit	ude	ofh	orizont	tal	load	at st	ati	on JI5		
		QYI5	(E1	0.3)	– M	agnit	ude	of v	ertica	11	oad a	t stat	ion	JI5		
		JM5		(15)	– N	umber	of	stat	ion							
		QXM5	(E]	0.3)	- M	agnit	ude	of h	orizont	ta l	load	l at st	ati	on JM5		

- QYM5 (E10.3) magnitude of vertical load at station JM5
- JL5 (I5) Number of station
- QXL5 (E10.3) Magnitude of horizontal load at station JL5
- QYL5 (E10.3) Magnitude of vertical load at station JL5
- IEND (A3) Enter "END" if last card in TABLE 5
- Remarks: A parabolic load distribution is assumed from station JI5 to station JL5, if all three values in the same direction are not zero;

A linear load distribution is assumed from station JI5 to station JM5, if only magnitudes of load at station JL5 are set to zero or left blank;

A uniform pressure distribution of load is assumed from station JI5 to station JL5, if QXI5 is set equal to QXL5 and all other magnitudes are set to zero or left blank;

A concentrated load is assumed at station JI5, if QXI5 and/ or QYI5 are/is not zero or blank, and all other magnitudes of load are zero or blank;

Except when no additional load is added to previous TABLE 5, QXI5 and QYI5 may never be both zero or left blank;

If the magnitude of load is zero at the beginning or end of a parabolic or linear distribution, specify a very small value, for instance, 1.0 E-20;

A horizontal load is positive from left to right, a vertical load is positive acting upwards, and a uniform pressure distribution is positive if acting outwards.

TABLE 6--DYNAMIC LOADS

Minimum of one card; maximum of twenty cards per run.

- No card is necessary if number of additional loadings (NADL in TABLE 1) is zero.
- Data in TABLE 6 is arranged in sets. A set may contain any number of cards, provided that the limit for the run is observed.
- If previous table is retained, a card may not be added to pre-existing set, but new sets (NADL in TABLE 1) of dynamic loadings may be added.

All remarks for TABLE 5, with appropriate changes, are valid here.

TABLE 6 (Continued)

JI6	QXIE	5 QYIO	6 JM6	QXM6	QYM6	JL6	QXL6	QYL6	IEND
[
1 :	D	15	25 30	4() 50	55	65	/5	/8-80
	Where	<u>e</u> :							
	JI6	(15)	- Number	of stat	ion				
	QXI6	(E10.3)	- Magnit	ude of h	orizonta	1 load	d at sta	tion JI6	
	QYI6	(E10.3)	- Magnit	ude of v	vertical	load a	at stati	on JI6	
	JM6	(15)	- Number	of stat	ion				
	QXM6	(E10.3)	- Magnit	ude of h	iorizonta	1 1 0a0	d at sta	tion JM6	
	QYM6	(E10.3)	- Magnit	ude of v	vertical	load a	at stati	on JM6	
	JL6	(15)	- Number	of stat	tion				
	QXL6	(E10.3)	- Magnit	ude of h	iorizonta	1 1 0a0	d at sta	tion JL6	
	QYL6	(E10.3)	- Magnit	ude of v	vertical	load a	at stati	on JL6	
	IEND	(A3)	- Enter	"END" if	last ca	rd in	the set		
	The r	next card if the d if LDTYI	d is nece dynamic 1 PE is not	ssary to oadings set equ	define are to b al to l	the ti e cons in TAB	ime func sidered 3LE 1.	tion, if a as pulses, See Figure	nd only i.e., 3.8.



Where:

FO (E10.3) - Maximum value of time function (peak)

TR (E10.3) - Time of rise in seconds

TD (E10.3) - Time of decay in seconds

TABLE 7--COLLAPSE PARAMETERS

Only for dynamic solution No card if dynamic solution is not required.

A) Maximum Displacements

One card


Where:

UMAX (E10.3) - Maximum allowable horizontal displacement

VMAX (E10.3) - Maximum allowable vertical displacement

<u>B) Maximum Shear</u>

Minimum of one card; maximum of ten cards



Where:

JS7 (I5) - Number of station where a change in allowable shear occurs. For uniform maximum allowable shear all over the structure, set JS7 equal to the last station and enter only one card

SMAXN (E10.3) - Ultimate shear at station JS7

Remarks: Ultimate shear is linearly interpolated between input stations;

All cards must be in ascending order of station numbers, starting with station 1;

Last card must contain the last station number of the structure.

C) Thrust-Moment Interaction Diagram

Minimum three cards; maximum twelve cards Multipliers (minimum one card, maximum ten cards)

JIA7	PMULN		BMULN	
6 10	21	30	40	

Where:

JIA7 (I5) - Number of station

PMULN (E10.3) - Thrust multiplier at station JIA7

BMULN (E10.3) - Moment multiplier at station JIA7

TABLE 7 (Continued)

Remarks: Multipliers are linearly interpolated between input stations;

1 7

All cards must be in ascending order of station numbers, starting with station 1;

Last card must contain the last station number of the structure;

For constant values for multipliers over the entire structure, set JIA7 equal to the last station and enter only one card.

Thrust-Moment Values (Two cards only; see Figure C.1)

First card--Thrust Values

P(2) P(3) ... P(J),J=2,8 ... 9 16 24 32 40 48 56 64

Second card--Moment Values



Where:

- P(J) (F8.0) Value to be multiplied by the thrust multiplier to obtain the value of thrust at point J of the thrustmoment interaction diagram
- M(J) (F8.0) Value to be multiplied by the moment multiplier to obtain the value of moment at point J of the thrustmoment interaction diagram

END QF RUN

5

One card. This is the last card in the data deck. It may be left blank or a termination message may be punched in columns 5 to 80 (alphameric). This message will be printed as the last line of the output.



APPENDIX D

•

PROGRAM DYNARCH: CODING LISTINGS AND PRINTOUT SHEETS

PROGRAM DYNARCH - MOUNTING FOR PROBLEM DTP1 COLUMNS # 3 5 1 2 4 6 7 8 12345673901234557890123456789012345678901234567890123456789012345678901234567890 //DYNARCH JOB (13446,238-82-2628,1), JERSON GUIMARAES', CLASS=B
/*PASSWORD GOIA **I*ROUTE PRINT HOLD** 11 EXEC FORTGCG,REGION.GO=148K // FORT.SYS IN DD * С FUNCTION FX С С C * С THIS FUNCTION DESCRIBES A SEMI-CIRCULAR ARCH C HAVING A DIAMETER OF 353.27 UNITS OF LENGTH * С * USED IN PROBLEM DTP1 C * * * * * * * * * * * * * * * * * * * С С FUNCTION FX (X) С IMPLICIT REAL * 8(A-H, O-Z) DATA D / 3.5327D 02 / FX = DSQRT (X * (D-X))**RETURN** END // GO.HEXIN DD * (INSERT DBJECT CODE DECK FOR MAIN PROGRAM AND SUBROUTINES) //GO.SYSIN DD * JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHDMA, APRIL 1974 DYNARCH TEST PROBLEMS - TWO HINGED CIRCULAR ARCH DF REINFORCED CONCRETE CIRC. ARCH - 180 DEGREES - STATIC SOLUTION ONLY - CENTRAL LOAD DTP1 YES 2 176.635 24 386.4 2 1 8. 8. 8. 0.3 2.2 2.2 4.1 7.9 9.8 9.8 11.7 12.0 25 8. 8. 8. END 0.3 2.2 7 2 1 6 1 1 2.2 11 19 23 1 1 7.9 9.8 4.1 1 12.0 24 1 9.8 28 1 11.7 30 1 25 2.0 1.0 10.0 1.0 2 E3 8.694E-2 1.0 -3.8 -2.88 0.0001 0.0002 0.0003 -1.5 -3.9 -4.0 0.0004 0.0005 3.0 E-3 1.75 E-4 1.0 E-3 -7.2 -2.4 -1.9 -1.35 -0.8 20. 21. 22. 23. 24. 1.0 E4 2.861E-1 -4.81 -4.84 -4.83 -4.82 -4.8 4.8 4.81 4.82 4.83 4.84 1.0 E-3 2.0 E-2 -10.6 -15-1 -19.6 -6.1 -1.6 1.6 6.1 10.6 15.1 19.6 1 11 222 1 25 1 1 END -1000. 25 END NORMAL TERMINATION OF EXECUTION 11

1 2 3 4 5 6 **7 8** 12345678901234567890123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - TWO HINGED CIRCULAR ARCH OF REINFORCED CONCRETE

PROBLEM DTP1

CIRC. ARCH - 180 DEGREES - STATIC SOLUTION ONLY - CENTRAL LOAD

2

TABLE 1. - CONTROL DATA

NO KEEP OPTIONS EXERCIZED

STATIC SOLUTION REQUIRED: YES

ACCELERATION OF GRAVITY

3.864D 02

AXIS OF STRUCTURE DESCRIBED BY A SMOOTH CURVE

SYMMETRICAL STRUCTURE AND LOADING

SOLUTION FOR HALF STRUCTURE

STATIC OUTPUT OPTION

SELF WEIGHT NOT INCLUDED

TABLE 9. - STATIC DISPLACEMENTS

STA. Ы v TH 0.0 0.0 0.0 1 -4.68130-03 1.2364D-04 3.9974D-04 2 -9.1621D-03 3 5.3413D-04 3.7639D-04 -1.3262D-02 4 1.17900-03 3.3964D-04 1.9779D-03 2.8293D-03 2.91380-04 5 -1.6831D-02 6 -1.9752D-02 2.3360D-04 7 -2.1943D-02 3.6182D-03 1.6828D-04 8 -2.3360D-02 4.2228D-03 9.74530-05 2.3187D-05 9 -2.39930-02 4.5223D-03 10 -2.3868D-02 4.40390-03 -5.24520-05 11 -2.3042D-02 3.7596D-03 -1.2739D-04 -2.1600D-02 2.5422D-03 +1.9955D-04 12 -1.96520-02 6.71000-04 -2.66860-04 13 -1.7324D-02 -1.8535D-03 -3.2730D-04 14 -1.4754D-02 -5.0462D-03 -3.7883D-04 15 -1.20860-02 -8.82430-03 -4.19500-04 15 -9.4595D-03 -1.3107D-02 -4.4738D-04 17 -7.00520-03 -1.7762D-02 -4.6058D-04 18 19 -4.8353D-03 -2.2621D-02 -4.5730D-04 20 -3.0364D-03 -2.7479D-02 -4.3580D-04 -1.66280-03 -3.20960-02 -3.94430-04 21 -7.2912D-04 -3.6209D-02 -3.3159D-04 22 -2.0445D-04 -3.9528D-02 -2.4581D-04 23 -7.0902D-06 -4.1748D-02 -1.3571D-04 24 -4.2558D-02 0.0 25 0.0

TABLE 10. - STATIC REACTIONS

STA.	HOR I ZONT AL	VERTICAL	NOMENT
1	6.3656D 02	1.0000D 03	0.0
25	-6.3656D 02	0.0	6.4196D 04

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 Dynarch test problems - Two Hinged Circular Arch of Reinforced Concrete

PROBLEM DT P1

CIRC. ARCH - 180 DEGREES - STATIC SOLUTION ONLY - CENTRAL LJAD

COMPLETE RESPONSE, TIME = 0.0

STA	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0	- 4 02500		6.0000D 00
2	-4.68130-03	1.2364D-04	-6.9759D (-6.05500)3		6.0000D 00
3	-9.1621D-03	5.3413D-04	-1.3165D (-5,35480	02 -1.05760 03	6.0000D 00
4	-1.3262D-02	1.1790D-03	-1.8542D (-4.65160)4	02 -1.09033 03	6.0000D 00
5	-1.6831D-02	1.9779D-03	-2.3083D	-3.92850	02 -1.11840 03	6.0000D 00
6	-1.97520-02	2.8293D-03	-2.6769D	-3.1586D 04	02 -1.1417D 03	6.0000D 00
7	-2.1943D-02	3.6182D-03	-2.9583D (-2.4351D	02 -1.1601D 03	6.0000D 00
8	-2.33600-02	4.2228D-03	-3.1515D (-1.6711D	02 - 1.1736) 03	6.0000D 00
9	-2.39930-02	4.5223D-03	-3.2555D (-8.9994D)4	01 -1.1820D 03	6.0000D 00
10	-2.3868D-02	4.4039D-03	-3.2700D	-1.2495D 04	01 -1.1854D 03	6.0000D 00
11	- 2. 3042D- 02	3.7696D-03	-3.1948D (· 6•5056D)4	01 -1.1836D 03	6.0000D 00
12	-2.1600D-02	2.5422D-03	-3.0302D (1.4233D	02 -1.1768D 03	6.0000D 00
13	-1.96520-02	6.7100D-04	-2.7771D	2.1900D	02 -1.1650D 03	6.0000D 00
14	- 1.73240-02	-1.86350-03	-2.4365D (2.9472D	02 -1.14820 03	6.0000D 00
15	-1-47540-02	+5,0462D-03	-2.03970 (3.6918D	02 -1.1265) 03	6.00000 00
16	-1.20860-02	-8.82430-03	-1.49880 (4.4207D	02 -1.0999D 03	6.00000 00
10	-0.45950-02	-1 31 070-02	-9 05740 0	5.1306D	02 -1.0686D 03	6 00000 00
11	-7.45750-03	1 77 (20, 02	- 3 33100 /	5•8185D	02 -1.0328D 03	6.00000 00
18	- 1.00520-03	-1.77020-02	-2.33130	6.4815D	02 -9.9253D 02	6.00000 00
. 19	-4.83530-03	+2.20210-02	5.1598U (7.1168D	02 -9.4801D 02	6.00000 00
20	-3.03640-03	-2.14180-02	1.33860	7.7216D	02 -8.9944D 02	6.00000 00
21	-1.6628D-03	-3.2096D-02	2.2311D (04 8•2933D	02 -8.4701D 02	6.0000D 00
22	-7.29120-04	-3.62090-02	3.1896D ()4 8•8295D	02 -7.9096D 02	6.0000D 00
23	-2.0445D-04	-3.9528D-02	4.2103D ()4 9.3279D	02 -7.3151D 02	6.0000D 00
24	-7.0902D-06	-4.1748D-02	5.2884D (9.7864D	02 -6.6894D 02	6.0000D 00
25 Normal	0.0 TERMINATION OF	-4.2558D-02 EXECJTION	6.4196D (04		6.0000D 00

PROGRAM DYNARCH -MOUNTING FOR PROBLEM DTP2.1 COLUMNS # 5 7 1 3 4 6 2 8 123456789012345678901234567890123456789012345678901234567890123455789012345578901234567890 //DYNARCH JOB (13446,238-82-2628,1),'JERSON GUIMARAES',CLASS=B /*PASSWORD GDIA /*ROUTE PRINT HOLD 11 EXEC FORTGCG, PARM. FORT = (NOSOURCE), REGION.GO=148K //FORT.SYSIN DD * С FUNCTION FX С С * * * * * * * č * THIS FUNCTION DESCRIBES A CIRCULAR ARCH HAVING A CENTRAL С * * Ĉ ANGLE OF OPENING OF 87.21 DEGREES, SPAN LO = 687 UNITS OF * С LENGTH, RISE OF 0.2 LO, USED IN PROBLEM DTP2.1 * ± č С * * * * * * * * * * * * * * С FUNCTION FX (X) С IMPLICIT REAL * 8(A-H, O-Z) DATA A, B, C / 841.575D00, 154.575D00, 360.675D00 / FX = DSQRT((A-X)*(B+X)) - CRETURN END GO.HEXIN DD # (INSERT OBJECT CODE DECK FOR MAIN PROGRAM AND SUBROUTINES) //GO.SYSIN DD * JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - LINEARLY ELASTIC DYNAMIC SOLUTION ONLY SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC D TP 2 P2.1 - CIRC. ARCH - 87.21 DEG. -NΩ 1 2 10 2 6 343.5 386.4 4.67E-2 1.555E-4 1 11.502 .504 11.502 0.318 0.795 0.795 4.795 11.365 15.365 15.365 15.842 16.16 7 11.502 .504 11.502 END 5 0.795 2 1 0.318 1 6 1 0.795 19 11 1 4.795 1 11.365 24 1 15.365 15.365 15.842 25 1 28 1 30 1 16.16 7 1 1.0 E4 .28618 -4.84 -4.83 -4.82 -4.81 -4.8 4.8 4.81 4.82 4.83 4.84 1.0 E-3 2.0 E-2 -10.6 -19.6 -15.1 -6.1 -1.6 1.6 6.1 10.6 15.1 19.6 11 1 222 1 7 .11 END 7 END -1.0 -1.0 1 1.0 E16 47.11 3. 4. E5 7 3.0 1.0 E5 1.0 E5 7 0.8 1.8 3.2 4.72 1.8 3.2 0-8 -4.5 -10.5 -16.5 -12. 12. 16.5 10.5 4.5 NORMAL TERMINATION OF EXECUTION //

1 2 3 4 5 6 7 8 12345678901234567890123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - LINEARLY ELASTIC DYNAMIC SOLUTION ONLY

PROBLEM DT P2

P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC

TABLE 2. - CROSS SECTION DESCRIPTION

CONTROL DATA

STA.	TOP FLANGE	WEB	BOT FLANGE
NO.	∦IDTH	Thickness	WIDTH
1	1.1502D 01	5.0400D-01	1.1502D 01

SEGMENT, MATERIAL, DEPTH DATA

S	EG	MAT	DEPT H	SEG	MAT	DEPTH	SEG	MA T	DEPTH
	2	1	3.180D-01	5	1	7.950D-01	6	1	7.9500-01
	11	1	4.795D 00	19	1	1.137D 01	24	1	1.537D 01
•	25	1	1.537D 01	28	1 -	1.584D 01	30	1	1.616D 01

CONTROL DATA

STA.	TOP FLANGE	WEB	BOT FLANGE
NG.	WIDTH	Thickness	WIDTH
7	1.1502D 01	5.04000-01	1.1502D 01

SEGMENT, MATERIAL, DEPTH DATA

SEG	MAT	DEPTH	SEG	MAT	DEPTH	S EG	MAT	DEPTH
2	1	3.180D-01	5	1	7.950D-01	6	1	7.950D-01
11	1	4.795D 00	19	1	1.137D 01	24	1	1.537D 01
25	1	1.537D 01	28	1	1.584D 01	30	1	1.616D 01

REINFORCEMENT DESCRIPTION

STA.	TOP REIN	FORCEMENT	BOTTOM REI	NFORCEMENT
NO.	DEPTH	AREA	DEPTH	AREA
7	0.0	0.0	0.0	0.0

TABLE 3. - STRESS-STRAIN CURVES

CURVE ND. 1

MATERIAL SPECIFIC WE	IGHT :	2 • 86 2D-0)1	
STRESS VALUE SCALE F	ACTOR	1.000D	04	
STRAIN VALUE SCALE F	ACTOR	1.000D-0	3	
STRESS INPUT VALUES				
-4.840 -4.830 -4.820 -	4.810 -4.8	800 4.	800 4.810	4.820

STRAIN INPUT VALUES -19.600-15.100-10.600 -6.100 -1.600 1.600 6.100 10.600 15.100 19.600

4.830 4.840

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - LINEARLY ELASTIC DYNAMIC SOLUTION ONLY

PROBLEM DTP2

P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC

۰.

TABLE 4. - SPECIFIED CONDITIONS

STA.	CODE	U	SPECIFIED DIS	PLACEMENT RDT
1	110	0.0	0.0	
1 7	HINGE 101	0.0		0.0

TABLE 5. - STATIC LOADS

NONE

TABLE 6. - DYNAMIC LOADING

DYNAMIC LOADING NO. 1

. Lander

INITIAL		1	INTERMEDIATE			FINAL		
STA.	×	Y	STA.	x	Y	ST A.	x	Y
1	-1.000D 00	0.0	0 0	.0	0.0	7 -1	00 00 00	0.0

PARAMETERS OF TIME FUNCTION

TIME OF RISING: TR = 0.0 TIME OF DECAY: TD = 1.0000D 16 MAXIMUM VALUE: FO = 4.7110D 01

TABLE 8. - STATION COORDINATES

STA.	X-COOR)	Y-COORD)
1	0.0		0.0	
2	4.8383D	01	4.0554D	01
3	1.01510	02	7.45620	01
4	1.58520	02	1.0178D	02
5	2.18500	02	1.2146D	02
6	2.80500	02	1.3340D	02
7	3.43500	02	1.3740D	02
	· · · · · · · · · · · · · · · · · · ·	•		

UNDIMENSIONAL CDEFFICIENTS FOR T/TO Y-DISPL(CR) THRUST(CR) MOMENT(1/4) MOMENT(CR) M(CROWN)-T 10 1.0000D 00 -1.9881D 00 -2.1439D-01 -6.1132D-01 8.9267D-01 8.9001D-01

COMPLETE RESPONSE, TIME = 1.5550D-02 MOMENT SHEAR CENTROID ST A. X DISPL Y DISPL THRUST 8.0800D 00 0.0 0.0 1 0.0 4.3189D 01 -5.3218D 03 8.0800D 00 -3.1226D-03 3.0450D-03 2.7265D 03 2 -8.6665D 02 -5.4756D 03 3 -6.0376D-03 6.7529D-03 -5.1986D 04 8.0800D 00 -7.3749D 02 -5.7079D 03 -6.0506D-03 5.6889D-03 -9.8544D 04 8.08000 00 4 8.0401D 02 - 5.7568) 03 -2.8474D-03 -5.5914D-03 -4.7786D 04 8.0800D 00 5 2.0443D 03 -5.4222D 03 6 -8.9693D-05 -2.2277D-02 8.1272D 04 8.0800D 00 9.9201D 02 -5.0306D 03 8.08000 00 -3.02140-02 1.43900 05 7 0.0

CONDUCTE DECONNEE TINE - 1 55500-02

PROBLEM DTP2 P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC

7.2584D-03 -1.4406D-02 4.0439D 03 8.0800D 00 2 9.5197D 02 -4.4634D 04 3 1.2166D-02 -2.8835D-02 6.4140D 04 8.0800D 00 2.2993D 02 -4.4247D 04 8.0800D 00 4 1.2015D-02 -3.6977D-02 7.8655D 04 -1.0209D 03 -4.4156D 04 8.0061D-03 -3.6389D-02 1.4209D 04 8.0800D 00 5 -6.6886D 02 -4.4287D 04 3.7114D-03 -3.3316D-02 -2.8015D 04 8.0800D 00 6 2.65200 01 -4.43570 04 -3.2334D-02 -2.6334D 04 8.0800D 00 7. 0.0 UNDIMENSIONAL COEFFICIENTS FOR Y-DISPL(CR) THRUST(CR) MOMENT(1/4) MOMENT(CR) M(CROWN)-T T/TO

5.00000-01 -2.12760 00 -1.89040 00 4.87930-01 -1.63360-01 -1.62940-01

P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC

SHEAR

THRUST

6.4060D 01 -4.4937D 04

CENTROID

8.0800D 00

COMPLETE RESPONSE, TIME = 7.7750D-03 STA. X DISPL Y DISPL MOMENT

0.0

PROBLEM DTP2

1

5

0.0

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES	
JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL	1974

DYNARCH TEST PROBLEMS - LINEARLY ELASTIC DYNAMIC SOLUTION ONLY

0.0

UNDIMENSIONAL CDEFFICIENTS FOR T/TO Y-DISPL(CR) THRUST(CR) MOMENT(1/4) MOMENT(CR) M(CROWN)-T 20 2.0000D 00 -8.9508D-01 -4.1664D-01 -4.2013D-01 5.1786D-01 5.1631D-01

ST A.	X DISPL	Y DISPL	MOMENT	SHEAR		THRUST	•	CENTROI	D
1	0.0	0.0	0.0				_	8.0800D	00
2	6.1592D-03	-8.4725D-03	1.1014D 05	1.7446D	03	-8.7858D	03	8.0800D	00
3	4.59850-03	-7.4104D-03	5.1187D 04	-9.3378D	02	-9.0069D	03	8.08000	00
4	6.8209D-04	-1.01790-03	-6.7725D 04	-1.8836D	03	-9.63290	03	8.0800D	00
5	-1-45440-04	-1-16220-03	-7-2621D 04	-7.7554D	01	-1.01250	04	8-08000	00
	E 07490-04	-9 92720-03	2 74190 04	1.5846D	03	-1.0071D	04	8.08000	00
-	5.01490-04	-0.92120-03	2.74100 04	8.8502D	02	-9.7761D	03	0.00000	00
7	0.0	-1.3603D-02	8.3479D 04					8.0800D	00

COMPLETE RESPONSE, TIME = 3.1100D-02

T/T0

PROBLEM DTP2 P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC

15 1.5000D 00 -2.0495D 00 -1.6812D 00 6.1810D-01 -5.0191D-01 -5.0047D-01

Y-DISPL(CR) THRUST(CR) MOMENT(1/4) MOMENT(CR) M(CROWN)-T

STA. X DISPL Y DISPL MOMENT SHEAR THRUST CENTROID 8.0800D 00 1 0.0 0.0 0.0 -1.4164D 03 -4.0196D 04 6.1541D-04 -5.8719D-03 -8.9416D 04 2 8.08000 00 1.1309D 03 - 3.9977D 04 5.5759D-03 -1.9676D-02 -1.8023D 04 8.0800D 00 3 1.8638D 03 -3.9400D 04 9.4652D-03 -3.5390D-02 9.9637D 04 8.0800D 00 -4 -8.7727D 01 -3.8979D 04 5 7.8316D-03 -4.0677D-02 9.4099D 04 8.0800D 00 -1.8365D 03 -3.9113D 04 3.5008D-03 -3.5171D-02 -2.1838D 04 8.0800D 00 6 -9.3572D 02 -3.9447D 04 7 -3.1148D-02 -8.0909D 04 0.0 8.0800D 00 UNDIMENSIONAL COEFFICIENTS FOR

P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC

COMPLETE RESPONSE, TIME = 2.3325D-02

PROGRAM DYNARCH

PROBLEM DTP2

ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES UNDER STATIC AND DYNAMIC LOADING JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - LINEARLY ELASTIC DYNAMIC SOLUTION ONLY

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES UNDER STATIC AND DYNAMIC LOADING JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - LINEARLY ELASTIC DYNAMIC SOLUTION ONLY PROBLEM DTP2 P2.1 - CIRC. ARCH - 87.21 DEG. - SYM. DYNAMIC UNIFORM PRESSURE - ELASTIC COMPLETE RESPONSE, TIME = 3.8875D-02 ST A. X DISPL Y DISPL MOMENT SHEAR THRUST CENTROID 1 0.0 0.0 0.0 8.0800D 00 -7.8326D 01 -3.3549D 04 5.5977D-03 -1.0968D-02 -4.9446D 03 8.08000 00 2 6.6052D 02 -3.3489D 04 8.0800D 00 9.8718D-03 -2.2716D-02 3.6753D 04 З 7.8121D 02 -3.3363D 04 1.0753D-02 -3.0948D-02 8.6070D 04 8.0800D 00 · 4 -3.9329D 02 -3.33333 04 5 7.5332D-03 -2.9914D-02 6.1242D 04 8.0800D 00 -1.5675D 03 -3.3573D 04 3.1128D-03 -2.1540D-02 -3.7712D 04 8.0800D 00 6 -9.7395D 02 -3.3912D 04 7. 0.0 -1.6457D-02 -9.9196D 04 8.0800D 00 UNDIMENSIONAL COEFFICIENTS FOR T/TO Y-DISPL(CR) THRUST(CR) MOMENT(1/4) MOMENT(CR) M(CROWN) - T 2.5000D 00 -1.0829D 00 -1.4452D 00 5.3393D-01 -6.1536D-01 -6.1356D-01 25 COMPLETE RESPONSE, TIME = 4.6650D-02 Y DISPL MOMENT SHEAR THRUST CENTROID STA. X DISPL 1 0.0 0.0 0.0 8.0800D 00 -2.7451D 02 -1.5013D 04 -1.5277D-03 -9.6132D-05 -1.7330D 04 8.0800D 00 2

-5.2348D 02 -1.5220D 04 -2.1266D-03 -1.4763D-03 -5.0378D 04 8.0800D 00 3 3.0504D 02 -1.5415D 04 -6.0257D-04 -7.6282D-03 -3.1120D 04 8.0800D 00 4 9.0641D 02 -1.5385) 04 5 1.1484D-03 -1.7017D-02 2.6101D 04 8.0800D 00 2.1914D 02 -1.5302D 04 8.0800D 00 1.1326D-03 -2.3580D-02 3.9935D 04 6 -1.0335D 02 -1.5289D 04 7 0.0 -2.55520-02 3.30950 04 8.0800D 00

UNDIMENSIONAL COEFFICIENTS FOR T/TO Y-DISPL(CR) THRUST(CR) MOMENT(1/4) MOMENT(CR) M(CROWN)-T 30 3.0000D 00 -1.6813D 00 -6.5159D-01 -1.9305D-01 2.0531D-01 2.0466D-01

FAILURE DID NOT OCCUR IN SPECIFIED TIME LIMIT ELAPSED TIME = 4.6805D-02

		PROGRA	M DYNARCI	H - EXAM	PLE OF D	ATA INPU	т	•	
COLUMNS	# 1	2	2	6		5	6	7	8
12345678	90123456	789012	34567890	1234567890	12345678	90123456	78901234	56789012	34567890
//GO. SYS	SIN DD *								
JERSON (DUARTE GU	IMARAE	S - OSU -	- STILLWAT	ER, OKLAN	HOMA, AP	RIL 1974		
DTP2	P2+2 -	WIDE F	LANGE ST	EEL BEAM -	ELASTIC	RESPONS	E ONLY -	SINUS.	TMPULSE
		•		NO	20/ /	1	1	2 60	
	1	2	10	240 • 11.502	.504	2.8	E=2 8.1 02	9065-5	
	-		0.318		0.795		0	• 795	
			4.795		11.365		15	• 365	•
	11		19.309	11.502	.504	11.5	02 EN	•10 D	
	2	1	0.318	5 1	0.795	6	1 0	.795	
	11 25	1	4.795	19 1 28 1	11.365	24	1 15	• 365 • 16	
	11	-	190909	20 1	191012	50			•
•	1		1054	20610	•		•	. `	•
-4.74	-4.73	-4.7	2 -4.7	•28818 L - 4•70	4.70	4.71	4.72	4.73	4.74
			1.0 E-3	2.0 E-2			_		
-15.7	-12.2	-8.6 1	4 -5.1	0 -1.57	1.57	5.10	8.64	12.2	15.7
	ī	222			•		•		
	11	1 1			0 (10	-	EN	D	
1. 5		-1.17	56 8		-0.618	11		-1.175	5 FND
-			10.	10.				••••	2.10
	11		1.0 E10	1 0 510					
	0.2	0.4	0.6	1.0 210	0.6	0.4	0.2		
-1.0	-1.0	-1.0	-1.0		1.0	1.0	1.0	1.0	
DTP3	RC BEAM	1 - COM KEEP	BINED ST	ATIC LOAD YFS	AND DOWN	WARD/UPW 2 2	ARD SINU	SOIDAL II 2 10	MPULSE
	1	2	10	90.	386.4	1.0	E-2 4.7	874E-5	
	1		0.3	8.	8.	8.	•	`	
			4.1		2•2 7•9		9	• 2	
			9.8		11.7		12	• 0	
	11	1	0 3	8.	8.	8. 7	EN 1 2	D 2	
	11	i	4.1	19 1	7.9	23	1 9	• 2	•
	24	1	9.8	28 1	11.7	30	1 12	•0	
	11 2				10.0	1.0			
	-		1.0 E3	0.0848433					
	-2.0	-4.0	-4.0	1 -4.0	0.001	0.002	0.003	0.004	0.005
-10.0	-6.25	-2.5	-2.0	-1.5	6.0	12.0	18.0	24.0	30.0
			1.0 E4	0.2861235					
-4.74	-4.73	-4.1	2 -4.7	L -4.70 2.0 F-2	4.70	4.71	4• 72	4.73	4.74
-15.7	-12.2	-8.6	4 -5.1	0 -1.57	1.57	5.10	8.64	12.2	15.7
1		-25.0			-25.0			-14 40	END
1 5		-14.6	9 8		- 22. 275	· 11		-25.0	END
1		1.0E	-15 3		7.725	5		14.69	
5		14.6	9 8	5.0	22.275	11		25.0	END
	11		3.0 E4	5.0					
	11		1.0 E5	1.0 E5	o -		a		
-6.6	1+2	2.0	-5.5	4.27	∠•> 5•3	1•1 7.7	6.5	4.35	
		NOR	MAL TERM	INATION OF	EXECUTI	ON		· · · •	
11	•	2	2			5	. 6	7	٥
• -		-				-		-	

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES

COMPLETE RESPONSE, TIME = 4.9144D-03

UNDER STATIC AND DYNAMIC LOADING

JERSON DUARTE GUIMARAES - DSU - STILLWATER, DKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP2 P2.2 - WIDE FLANGE STEEL BEAM - ELASTIC RESPONSE ONLY - SINUS. IMPULSE

ST A.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	6.6522D-04	C.O	0.0		/	8.0800D 00
2	5.3330D-04	-7.97410-02	1.2682D 05	5.28420 03	1.76370 01	8.08000 00
3	4.0785 D-04	-1.5747D-01	2.4426D 05	4.89320 03	1.3224D 01	8.0800D 00
4	2.9448D-04	- 2.31310-01	3.5401D 05	4.5729D 03	6.8777D 00	8.08000 00
5	1.97640-04	-2-99520-01	4.64050.05	4.5853D 03	2.9338) 00	8-08000 00
	1 20540-04	-3 (03/0-01	5 (3330 05	4.1382D 03	-5.2034D-01	
	1.20540-04	-3-80340-01	5.65570 05	3.29510 03	-3.26130 00	8.08000 00
7	6.4383D-05	-4.12210-01	6.4245D 05	2.50300 03	-4.5659D 00	8.0800D 00
8	2.8097D+05	-4.5387D-01	7.0252D 05	1.8253D 03	-6.2314D 00	8.0800D 00
9	8.5481D-06	-4.84350-01	7.4533D 05	1,24110 03	-7-37970 00	8.0800D 00
10	1.1024D-06	-5.02960-01	7.7612D 05	4 95250 02	~9 09140 00	8.0800D 00
11	0.0	-5.09220-01	7.8800D 05	4 077290 02	-9409140 00	8.0800D 00

PROBLEM DTP2

P2.2 - WIDE FLANGE STEEL BEAM - ELASTIC RESPONSE ONLY - SINUS. IMPULSE

COMPLETE RESPONSE, TIME = 1.5071D-02

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	4.9012D-03	0.0	0.0	1 00050 0/	1 00 700 00	8.0800D 00
2	3•9292D-03	-2.16420-01	3.3180D 05	1.38250 04	1.2273) 02	8.0800D 00
3	3.0039D-03	-4.2756D-01	6.6075D 05	1.3706D 04	1.1099D 02	8.0800D 00
4	2,16810-03	-6-28200-01	9.76130 05	1.3141D 04	9.3120D 01	8-08000 00
				1.1998D 04	6.82030 01	
5	1.42640-03	-8.13310-01	1.20410 00	1.0538D 04	4.0326D 01	8.08000 00
6	8.90360-04	-9.7832D-01	1.5170D 06	8.9746D 03	8.3857D 00	8.0800D 00
7	4.77120-04	-1.1192D 00	1.7324D 06	7.4541D 03	-2.2683D 01	8.0800D 00
8	2.0883D-04	-1.2325D 00	1.9113D 06	5 34200 03	-5 05753 01	8.0800D 00
9	6.3951D-05	-1.31550 00	2.0395D 06			8.0800D 00
10	8.5641D-06	-1.3660D 00	2.1093D 06	2.90/50 03	-1.22330 01	8.0800D 00
11	0.0	-1.3829D 00	2.1319D 06	9.4280D 02	-8.2378D 01	8.0800D 00

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES

COMPLETE RESPONSE, TIME = 2.5227D-02

UNDER STATIC AND DYNAMIC LOADING JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

.

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	8.0624D-03	0.0	0.0			8.0800D 00
2	6.46160-03	-2.77730-01	4.3601D 05	1.8167D 04	1.9708D 02	8.0800D 00
2	4 93900-03	-5 49520-01	P 55130 05	1.74630 04	1.62570 02	8:08000.00
2	+++++++++++++++++++++++++++++++++++++++	-9,48920-01	8.55150 05	1.6319D 04	1.2414D 02	8.08000 00
4	3.5647D-03	-8.05710-01	1.2468D 06	1.5326D 04	8.29130 01	8.0800D 00
5	2.3935D-03	-1.04310 00	1.6146D 06	1 37760 06	(17050 01	8.0800D 00
6	1.4612D-03	-1.2548D 00	1.9452D 06	1.57760 04	4.17950 01	8.0800D 00
7	7.80770-04	-1.4355D 00	2.2229D 06	1.1570D 04	6.96323 00	8.08000 00
0	3 30680 06	1 59000 00	2 ((2 7 0 0 (9.2003D 03	-2.07160 01	
8	3.39680-04	-1.58090 00	2.44310 08	6-8599D 03	-4.2404D 01	8.08000 00
9	1.01910-04	-1.6874D 00	2.6086D 06	4.40600 03	-5.70930 01	8.0800D 00
10	1 • 19690- 05	-1.7525D 00	2.7143D 06			8.0800D 00
11	0.0	-1.7744D 00	2.7514D 06	1.5454D 03	-6.3806D 01	8.0800D 00

P2.2 - WIDE FLANGE STEEL BEAM - ELASTIC RESPONSE ONLY - SINUS. IMPULSE

PROBLEM DTP2

PROBLEM DTP2

P2.2 - WIDE FLANGE STEEL BEAM - ELASTIC RESPONSE ONLY - SINUS. IMPULSE

COMPLETE RESPONSE, TIME = 2.6538D-02

STA.	X DISPL	Y DI SPL	MOMENT	SHEAR	THRUST	CENTROID
1	8.0970D-03	0.0	0.0	1 81200 04	2 04380 02	8.0800D 00
2	6.4860D-03	-2.78640-01.	4.3509D 05	1 75140 04	1 00010 02	8.0800D 00
3	4.95380-03	-5.50350-01	8.5548D 05	1 44210 04	1 49790 02	8.0800D 00
4	3.57100-03	-8.08470-01	1.2546D 06	1.63710.04	1.00700 02	8.0800D 00
5	2.3939D-03	-1.0466D 00	1.6235D 06	1.99710 04	1.44510 02	8.0800D 00
6	1.4581D-03	-1.25900 00	1.9507D 06	1.36310 04	1.15610 02	8.0800D 00
7	7.75920-04	-1.4403D 00	2.2290D 06	1.15950 04	8.75313 01	8.0800D 00
8	3.3446D-04	-1.5862D 00	2.4538D 06	9.36980 03	5.81470 01	8.0800D 00
9	9.7715D-05	-1.6930D 00	2.6201D 06	6.9285D 03	3.4606D 01	8.0800D 00
10	9•7430D-06	-1.7582D 00	2.7207D 06	4.19170 03	1.65500 01	8.0800D 00
11	0.0	-1.78010 00	2.7543D 06	1.3996D 03	8.02720 00	8.0800D 00

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP2 P2.2 - WIDE FLANGE STEEL BEAM - ELASTIC RESPONSE ONLY - SINUS. IMPULSE

COMPLETE RESPONSE, TIME = 2.7193D-02

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	8.0962D-03	0.0	0.0			8.0800D 00
2	6.4874D-03	-2.7841D-01	4.3242D 05	1.8017D 04	1.9778D 02	8.0800D 00
٦	4-95660-03	-5,49950-01	8-52910 05	1.7520D 04	1.70740 02	8.08000 00
,	2 57/50 03	0.07000.01		1.6638D 04	1.4068D 02	
4	3.57450-03	-8.07920-01	1.25220 06	1.5326D 04	1.0714D 02	8.08000 00
5	2.3972D-03	-1.0460D 00	1.6200D 06	1.3831D 04	7.5588D 01	8.0800D 00
6	1.4606D-03	-1.2583D 00	1.9520D 06	1 17490 04	6 50503 01	8.0800D 00
7,	7.7756D-04	-1.4395D 00	2.2339D 06	1.11490 04	4. 000 01	8.0800D 00
8	3.35830-04	-1.5853D 00	2.4540D 06	9.1706D 03	2.1805D 01	8.0800D 00
9	9.87720-05	-1.69200.00	2-61600 06	6.7468D 03	3.5994D 00	8-08000 00
,		. 75700 00		4.2090D 03	-6.3270D 00	
10	1.09040-05	-1.75700 00	2.11100 06	1.4457D 03	-1.05310 01	8.0800D 00
11	0.0	-1.7789D 00	2.7517D 06			8.0800D 00

COMPLETE RESPONSE, TIME = 2.8012D-02

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	8.0669D-03	0.0	0.0	·		8.0800D 00
2	6.4681D-03	-2.7754D-01	4.2690D 05	1.7787D 04	1.92310 02	8.0800D 00
2	4.9458D-03	-5.4829D-01	8.47050 05	1.7506D 04	1.59960 02	8 08000 00
5	4. 94900 09	5.40250 01	0.41050 05	1.6732D 04	1.2195D 02	0.00000 00
4	3.5706D-03	-8.0557D-01	1.2486D 06	1.5383D 04	7.9299D 01	8.0800D 00
5	2.3987D-03	-1.0430D 00	1.6178D 06	1 36740 04	2. 4.9420.01	8.0800D 00
6	1.4661D-03	-1.25470 00	1.9460D 06	1.30740 04	514802U UI	8.0800D 00
7	7.8509D-04	-1.4354D 00	2.2263D 06	1.16800 04	-1.25110 01	8.08000 00
	2 4226 0 04	1 59090 00		9.2948D 03	-5.2491D 01	
8	3.43300-04	-1.58080 00	2.44940 06	6.7659D 03	-8.6388D 01	8.08000 00
9	1.0486D-04	-1.6872D 00	2.6118D 06	4.02020 03	-1.1126D 02	8.0800D 00
10	1. 37 72D- 05	-1.7520D 00	2.7083D 06	1 2/250 02	1 2//20 02	8.0800D 00
11	0.0	-1.7738D 00	2.7386D 06	1.20250 03	-1.24430 02	8.0800D 00

FAILURE DID NOT OCCUR IN SPECIFIED TIME LIMIT ELAPSED TIME = 2.8012D-02

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES UNDER STATIC AND DYNAMIC LOADING JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM PROBLEM DTP3 RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSDIDAL IMPULSE TABLE 1. - CONTROL DATA RETAIN PRIOR DATA TABLES 4, STATIC SOLUTION REQUIRED: YES ACCELERATION OF GRAVITY 3.864D 02 AXIS OF STRUCTURE DESCRIBED BY A SMOOTH CURVE SYMMETRICAL STRUCTURE AND LOADING SOLUTION FOR HALF STRUCTURE STATIC OUTPUT OPTION 2 SELF WEIGHT NOT INCLUDED NUMBER OF DYNAMIC LOADINGS 2 DYNAMIC OUTPUT OPTION 2 OUTPUT INTERVAL 10 TIME LIMIT 1.000D-02 TYPE OF DYNAMIC LOADING: IMPULSE

INELASTIC RESPONSE REQUIRED

TIME INTERVAL 4.787D-05

TABLE 7. - COLLAPSE PARAMETERS

.

DISPLACEMENT LIMITS

MAXIMUM HORIZONTAL DISPL: 1.0000D 00 MAXIMUM VERTICAL DISPL: 5.0000D 00

SHEAR	LIMITS	
	TERM	SHEAR
	STA.	VALUE

11 3.0000D 04

INTERACTION DIAGRAM DATA

TERM STA.	AXIAL FORCE MULTIPLIER	MUL	IOMENT TIPLIER			
11	1.0000D 05	1.0	000D 05			
AXIAL FORCE 0.0 1.2	INPUT VALUES 00 2.000 3.050	4.270	2.500	1.100	0.550	0.0
MOMENT INPUT -6.600 -6.6	VALUES 00 -6.600 -5.500	0.0	5.300	7.700	6.500	4.350

. •

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP3

RC BEAM - COMBINED STATIC LDAD AND DOWNWARD/JPWARD SINUSOIDAL IMPULSE

CCMPLETE RESPONSE, TIME = 0.0

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0	2 12750 0	2 2 770/ 0 01	6.3851D 00
2	0.0	-1.5680D-02	1.9235D 04	2.13/50 0	3 -2.11860-01	6.3851D 00
3 ·	0.0	-3.09170-02	3.6446D 04	1.9125D 0	3 -8.0433D-01	6.3851D 00
4	0.0	-4.5310D-02	5.1632D 04	1.6875D 0	3 -1.27233 00	6.3851D 00
5	0.0	-5.8502D-02	6.4793D 04	1.4625D 0	3 -1.6818D 00	6.3851D 00
6	0.0	-7.0185D-02	7.5930D 04	1.23750 0	3 -2.0328D 00	6.3851D 00
7.	0.0	-8.0100D-02	8.5041D 04	1.01250 0	3 -2.3253D 00	6.3851D 00
8	0.0	-8.8033D-02	9.2128D 04	7.8750D 0	2 -2.55920 00	6.3851D 00
9	0.0	-9-3818D-02	9.71900 04	5.6250D 0	2 -2.7347D 00	6.38510 00
10	0.0	-9.73360-02	1.00230-05	3.3750D 0	2 -2.8517D 00	6.38510.00
11	0.0	-0.95160-02	1 01 24 0 05	1.1250D 0	2 - 2.91020 00	4 39510 00
11	0.0	-3.02100-02	1.01240 05			0.20210 00

PROBLEM DTP3

.

.

RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSDIDAL IMPULSE

÷.,

· .

COMPLETE RESPONSE, TIME = 4.7874D-04

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	1.04700-03	0.0	0.0	4 12240 02	1 72020 02	6.3851D 00
2	5.1019D-04	-1.00990-01	3.71120 04	4.12500 05	1.72050 05	6.3851D 00
3	1.3616D-04	-2.0112D-01	1.0652D 05	7.71130 03	3.40750 03	4.5713D 00
4	-1.3075D-04	-2.96050-01	1.3720D 05	3.4097D 03	4.3837D 03	4.3271D 00
5	-2.1561D-04	-3.83910-01	1.8430D 05	5.2322D 03	5.09863 03	4.2287D 00
6	-2.35210-04	-4.6214D-01	2.2505D 05	4.5280D 03	4.9160D 03	4.1507D 00
7	-1.9653D-04	-5.28510-01	2.6571D 05	4.5183D 03	4.0385D-03	4.1507D 00
8	-1-58020-04	-5,8087D-01	2.6967D 05	4.3982D 02	2.9972D 03	4.1507D 00
۰ ۵	-1-05940-04	-6-19020-01	2.80540 05	1.2079D 03	2.02560 03	4.1507 D 00
10	-5.54410-05	-6 42390-01	2 95550 05	1.6668D 03	1.3390D 03	4 15070 00
10	-9.94610-09		2.77790 09	2.2693D 01	9.8060D 02	4.15070 00
11	0.0	-0.20190-01	2.90 100 00			4.12010 00

JERSON DUARTE GUIMARAES - DSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP3

RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/UPWARD SINUSDIDAL IMPULSE

COMPLETE RESPONSE, TIME = 9.57480-04

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	5.1613D-03	0.0	0.0			6.3851D 00
2	3.2442D-03	-1.8614D-01	6.3489D 04	7.0543D 03	1.6440D 03	6.3851D 00
-	1 50970-03	-3 70800-01	1 47800 05	1.1600D 04	4.43250 03	4 22870 00
5	1.38810-03	-3.10800-01	1.87890 05	9.1537D 03	6.3794D 03	4.22810 00
4	3.1218D-04	-5.4670D-01	2.5028D 05	1.23020 04	7.51220 03	4.1507D 00
5	-6.1730D-04	-7.0941D-01	3.6100D 05	2 02250 0		4.0982D 00
6	-1.2061D-03	-8.53030-01	3.9541D 05	3.82350 02	5 7.41940 03	4.0220D 00
. 7	-1 53460-03	-9 74900-01	4.0639D 05	1.2195D 03	5.01730 03	4.03820.00
4.	-1.55400-05		4.03390 03	1.5392D 03	-1.4060D 02	4.03820 00
8	-1.5777D-03	-1.0733D 00	4.2024D 05	9.93990 02	-4.67010 03	3.9894D 00
9	-1.0226D-03	-1.1447D 00	4.2919D 05	1 20750 00		3.9694D 00
10	-4.8961D-04	-1.1870D 00	4.2794D 05	-1.38750 02	2 -6.21450 03	3.9781D 00
11	0.0	-1 20110 00	4 31 7 40 05	4.2167D 02	2 -6.17670 03	A 01490 00
* *	0.0	-1.20110 00	4032140 0 0			HOUTHOU 00

COMPLETE RESPONSE, TIME = 1.24470-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	9.40500-03	0.0	0.0			6.3851D 00
2	6.2905D-03	-2.3848D-01	9.2311D 04	1.0257D 04	1.9535D 03	5.1881D 00
-	3 44440-03	-6 73300-01	2 17470 05	1.3907D 04	3.7951D 03	A 1507D 00
3	5.44040-05	-4.13300-01	2.11410 05	1.3937D 04	3.5047D 03	4.19010 00
4	1.0924D-03	-6.9666D-01	3.4292D 05	5.3468D 03	1.7859D 03	4.15070 00
5	-1.34280-03	-9.0195D-01	3.9104D 05	5 20220 02	4 14442 02	4.0982D 00
6	-2.94890-03	-1.08660 00	4.3877D 05	5.30320 03	~4.1044J 03	4.03350 00
7	-3 03120-03	-1-24260 00	4-45460 05	7.44010 02	-1.0427D 04	4-02520 00
'	-3.35120-03	-1.24200 00	+ • + 5 + 6 0 0 5	7.7950D 01	-1.1114D 04	402320 00
8	-4.1792D-03	-1.3683D 00	4.4617D 05	5.9908D 02	-1.13710 04	3.9894D 00
9	-2.4546D-03	-1.4598D 00	4.5156D 05	(05 (0D 03	-1 27752 04	3.9694D 00
10	-1.05520-03	-1.5139D 00	4.57820 05	6.9549D UZ	-1.3775J 04	3.9781D 00
11	0.0	-1 53150 00	4 56800 05	-1.1352D 02	-1.58790 04	3. 991 50 .00
TT.	U • U	-1.00100	4.3000 03			2. 2. 2. 20 00

.

• •

FAILURE DUE TO THRUST-MOMENT INTERACTION AT STATION 10

FAILURE OCCURRED AT TIME = 1.2447D-03 SECS.

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP3 RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSDIDAL IMPULSE

SOLUTION FOR DYNAMIC LOADING NO. 2

STA.	POINT	DYNAMIC	LOADS	STATIC	LOADS	DEPTH OF
NO.	MASS	HORI ZONTAL	VERTICAL	HOR I ZONT AL	VERTICAL	CENTROID
1	9.7200D-02	0.0	6.0375D 00	4.0014D 00	-1.1250D 02	6.3851D 00
2	1.9440D-01	0.0	3.54750 01	4.0558D-02	-2.2498D 02	6.3851D 00
3	1.9440D-01	0.0	6.9382D 01	-7.1337D-02	-2.2498D 02	6.3851D 00
4	1.9440D-01	0.0	1.0158D 02	-1.4533D-01	-2.2498D 02	6.3851D 00
5	1.9440D-01	0.0	1.3201D 02	-1.8616D-01	-2.2498D 02	6.3851D 00
6	1.9440D-01	0.0	1.5942D 02	-1.9856D-01	-2.24980 02	6.3851D 00
7	1.9440D-01	0.0	1.8217D 02	-1.8728D-01	-2.2498D 02	6.3851D 00
8	1.94400-01	0.0	2.0007D 02	-1.5704D-01	-2.2498D 02	6.3851D 00
9	1.9440D-01	0.0	2.1310D 02	-1.1259D-01	-2.2498D 02	6.3851D 00
10	1.9440D-01	0.0	2.2128D 02	-5.8564D-02	-2.2498D 02	6.3851D 00
11	9.7200D-02	0.0	1.1215D 02	0.0	-1.1249D 02	6.3851D 00

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES UNDER STATIC AND DYNAMIC LOADING

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP3

RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSDIDAL IMPULSE

COMPLETE RESPONSE, TIME = 4.7874D-04

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	3.6898D-04	0.0	0.0			6.38510 00
2	1.0785D-04	7.0289D-02	-6.58350 04	-7.3149D 03	1.01920 03	6.3851D 00
3	-1.14420-05	1.39030-01	-6.2090D 04	4.1601D 02	3.2192D 03	6.7568D 00
	2020.020 0F	2 05000 01	-7 28(30 02	6.0781D 03	3.2969) 03	1 01380 01
4	- 8. 32 020+05	2.05990-01	-1.50850 05	-1.8605D 03	2.8108D 03	1.01380 01
5	-6.0838D-05	2.6586D-01	-2.4132D 04	1.9366D 03	2.53350 03	8.9662D 00
6	-6.8263D-05	3.21920-01	-6.70170 03	1.21720 02	1.37910 03	1.0302D 01
7	-3.1999D-05	3.6869D-01	-5.6062D 03	1. ((110.00		1.0447D 01
8	-1.8467D-05	4.0457D-01	-6.92120 03	-1.46110 02	8.86310 02	1.03020 01
9	-1.23670-05	4.3084D-01	-6.6661D 03	2.8344D 01	5.0519D 02	1.0302D 01
,	(() 0) 0 0(((79(0 0)		6.9538D 01	2.4930D 02	1 06670 01
10	-0+41810-06	4•41800+01	-0.02720 03	-1.7587D 02	1.2374) 02	1+04470 UI
11	0.0	4.5315D-01	-7.6221D 03			1.0302D 01

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DTP3

.

RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSDIDAL IMPULSE

COMPLETE RESPONSE, TIME = 9.5748D-04

STA.	X DISPL	Y DISPL	MOMENT	SHEAR		THRUST	CENTROID
1	3.0897D-03	0.0	0.0				6.3851D 00
2	1,92350-03	1.5202D-01	6.59380 04	7.3263D	03	1.9129) 03	6.38510 00
-	10,2000 00			-3,3967D	02	3.9082D 03	0190910 00
3	5 .57 09D-04	3.05580-01	6.2881D 04	- 8 1 (3 0 0	<u>^</u>	E 03/00 03	6.3851D 00
4	-2.3535D-04	4.6061D-01	-1.05910 04	-0.10300	05	5.95490 03	1.0626D 01
_				4.)897D	03	6.4650D 03	
5	-4.1003D-04	5.8744D-01	2.62200 04	-3.94910	03	6. 56267 03	7.0095D 00
6	-1.3413D-03	7.1496D-01	-9.3215D 03		05	0. 50205 05	1.0626D 01
-	/			4.9258D	01	5.4395D 03	
ι.	-5.93290-04	8-11080-01	-8.8781D 03	1,2155D	02	4.4849D 03	1.06260 01
8	-6.25410-04	8.9675D-01	-7.7841D 03	3			1.0626D 01
•	- 2 45050-04	0 66100-01	- 0 26020 03	-5.3900D	01	2.8968D 03	1 05 (00 01
7	- 3.43930- 04	9.55100-01	-0.20920 03	-1.8033D	02	2.2548) 03	1.02080 01
10	-1.9268D-04	9 .9 2950-01	-9.8922D 03	8			1.0626D 01
11	0.0	1.00450.00	-9-29840 03	6.5982D	01	1.6400D 03	1.05680.01
**		2,00,00,00		-			1100000 01

PROBLEM DTP3

RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSOIDAL IMPULSE

COMPLETE RESPONSE, TIME = 1.9150D-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	1.8551D-02	0.0	0.0	6 00160 03	1 25100 02	6.3851D 00
2	1.28960-02	3.1945D-01	4.32150 04	4.80180 03	1.55100 05	6.3851D 00
3	7.3652D-03	6.39900-01	4.3402D 04	2.07510 01	4.04600 03	6.3851D 00
4	1.81960-03	9.61360-01	-1.7304D 04	-6./449D 03	6.7478D 03	1.0646D 01
5	- 1.7735D-05	1.2366D 00	-5.5818D 03	1.30210 03	8.13800 03	1.0626D 01
6	-3.90930-03	1.4969D 00	-1.6110D 04	-1.1698D 03	8.22480 03	1.0646D 01
7	-3.79010-03	1.7142D 00	-1.8944D 04	-3,1475D 02	8.7852D 03	1.0646D 01
8	+5.08060-03	1.88080 00	-1.63520 04	2.87950 02	9.5752D 03	1.0646D 01
0	-3.05800-03	2 00360 00	-1 69470 04	-6.60560 01	1.1154D 04	1.06460.01
,	- 3. 03870- 03	2.00300.00	1 02200 04	-2.5366D 02	1.1663D 04	1.04400 01
10	-2.20210-03	2.08100 00	-1.92300 04	-2.0937D 01	1.2625D 04	1.0046D UI
11	0.0	2.1070D 00	-1.9419D 04			1.06460 01

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH TEST PROBLEMS - SIMPLY SUPPORTED HORIZONTAL BEAM

PROBLEM DT P3

RC BEAM - COMBINED STATIC LOAD AND DOWNWARD/JPWARD SINUSOIDAL IMPULSE

COMPLETE RESPONSE, TIME = 3.3512D-03

CENTROID
6.3851D 00
6.3851D 00
1.06460 01
1.0646D 01
1.0646D 01
1 06460 01
1.08480 01
1.0646D 01
1.06460 01
1,06460 01
1.0646D 01
1.0646D 01

FAILURE DUE TO VERTICAL DEFLECTION AT STATION 11

FAILURE OCCURRED AT TIME = 4.4523D-03 SECS.

COMPLETE RESPONSE, TIME = 4.4523D-03

STA	X DISPL	Y DISPL	MOMENT	SHEAR	THRUS	т	CENTROID)
1	1.5204D-01	0.0	0.0				6.38510	00
2	1.1970D-01	7.6205D-01	1.5383D 04	1.7092D	03 1.6482D	03	6.3851D	00
а	8.7669D-02	1-52450 00	-7.3600D 03	- 2.5269D	03 5.61950	03	1.0646D	01
				-1.1211D	03 8.3523D	03	1000400	•1
4	5.8849D-02	2.2674D 00	-1.74520.04	-1.9646D	03 1.1856D	04	1.0646D	01
5	3.31300-02	2.9637D 00	-3.5136D 04	-2 13400	02 1 44770	04	1.0646D	01
6	1.61150-02	3.5664D 00	-3.7058D 04	-2413490	02 1.44770	04	1.0646D	01
7	3.6428D-03	4.0700D 00	-3.3955D 04	3.4466D	02 1.67645	04	1.0646D	01
	1 57/30 03	4 49 30 0.00	4 37140 04	-1.0838D	03 1.8764D	04		01
8	-1.07450-05	4.48290 00	-4.57140 04	-4.0076D	01 1.85570	.04	1.06460	01
9	-4.6350D-03	4.7786D 00	-4.4075D 04	- 2.77020	01 1.93700	04	1.0646D	01
10	-2.4938D-G3	4.9559D 00	-4.4324D 04			•••	1.0646D	01
11 Normal	0.0 Termination of	5.0144D 00 Execution	-4.3525D 04	8.8782D	01 1.88630	04	1.0646D	01

	S #	PROGRAM	M DYNARCH	4 - E)	CAMP	LE OF D	ATA INPU	т		
1 22/5/	1 1 1	2	3	22/5/70	4	2245470	5	6	7	8
123456	1890123450	578901Z:	345678901	2345678	1901	2345678	90123456	18901234	+56789012	34567890
//GO.SN JERSON DYNARCH DAP1	YSIN DD * DUARTE GU A APPLICAT CIRCUL	JIMARAES FION PRO AR ARCH	S - OSU - OBLEMS - - 180 DE	- STILLY TWO-HIN EG SI	NATE NGEI M. Kes	ER, OKLA D CIRCUL DYN. LO 1	HƏMA, AP Ar Arch Ad And S 2 1	RIL 1974 OF REINF ELF WEIG	- - ORCED CO - ORCED C	NCRETE
	1	2	12	177.		386.4	1.2	E − 2 1	L-2 E-4	
	- - :		0.3	8.		2.2	ö.		2.2	
			4.1			7.9		9 13	8.8	-
	13		7 • 0	8.	•	8.	8.	EN	ND ND	
	2	1	0.3	6 19	1	2.2	23		2.2	
	24	1	9.8	28	i	11.7	30	1 12	2.0	
	13		2.0	1.0		10.0	1.0			
	۲		1.0 E3	8.6948	E-2					
-1.5	-3.9	-4.0	-3.8	-2.88	3 =-3	0.0001	0.0002 -4	0.0003	3 0.0004	0.0005
-7.2	-2.4	-1.9	-1.35	-0.8		20.	21.	22.	23.	24.
-4.84	-4.83	-4.82	1.0 E4	2.861E	-1	4.8	4.81	4.82	4-83	4.84
4.04	4005	4002	1.0 E-3	2.0 E	-2	110	4001	4002	4000	44.04
-19.6	-15.1	-106	-6.1	-1.6		1.6	6.1	10.6	15.1	19.6
	ī	222								
1	13 1-0E-15	1 1						EI	ND	END
11	1.0 E-15	-1.0'E	-15 12	7071067	8 -	.707106	78 13	1.0	-1.0	END
		i	2400.	5.0 E	-3	1.0 E	-2			
	13		3.0 E5							
	13	1.8	1.0 E5	1.0	, E5	3.2	1.8	0.8		
-4.5	-10.5	-16.5	-12.			12.	16.5	10.5	4.5	
DAP2	CIRC. /	ARCH - 3	180 DEG.	- NONS	ES (DYN. L	DAD AND 2 1	SELF WEI	GHT - IN 2 10	ELASTIC
	1	1	24	354.		386.4	2.0	E-2		
	1		0.3	8.		8. 2.2	8.	2	2.2	
			4.1			7.9	•	ç	.8	
	25		9.8	8.		8.	8.	L Z EN	2•0 . 1D	
	2	1	0.3	6	1	2.2	7	1 2	2.2	
	24	1	4•1 9•8	28	1	7.9	23 30		2.0	
	25		2.0	1.0		10.0	1.0			
	1	222		•						
	25	222						-		
1	25	-1.0 E	-15					E	ND .	
25		-1.0 E	-15		• •	30310/				END
5	1.0 E-15 1.0	-1.0 E-	-15 6. 8,	.7071067	18 - 18 -	.707106	78 7 78 9	1.0 1.0 E-1	-1.0 L5 -1.0 E	-15 END
·		4	4000.	5.0 E	-3	1.0 E	-2			
	25	3	3.0 E5	4.						
	25	1 0	1.0 E5	1.0	,E5	2 2	1 9	0 9		
-4.5	-10.5	1.8 -16.5	-12.	4 • 1 4	5	12.	16.5	10.5	4.5	
		NORI	MAL TERM	INATION	OF	EXECUTI	ON			
1	1	2	3		4		5	6	7	8
123456	7890123450	5789012	34567890	12345678	3901	2345678	90123456	78901234	56789012	34567890

. •

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES

PROBLEM DAP1

TABLE 1. - CONTROL DATA

UNDER STATIC AND DYNAMIC LOADING

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974

DYNARCH APPLICATION PROBLEMS - TWO-HINGED CIRCULAR ARCH OF REINFORCED CONCRETE

CIRCULAR ARCH - 180 DEG. - SYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

3.864D 02

1

2

10

155

TYPE OF DYNAMIC LOADING: FORCING PULSE INELASTIC RESPONSE REQUIRED TIME INTERVAL 1.2000-04 TABLE 5. - STATIC LOADS -----THITCOMEDIATE

1.200D-02

	101111	AL .	1	INTERMEDIATE				FINAL	
STA.	x	Y	STA.	x	· Y	STA.	x	Y	
1	1.0000-15	0.0	0 0	•0	0.0	0	0.0	0.0	

TABLE 6. - DYNAMIC LOADING

DYNAMIC LOADING NO. 1

INTERMEDIATE INITIAL FINAL Y STA. STA. STA. x Y х X v 11 1.300D-15-1.000D-15 12 7.071D-01-7.071D-01 13 1.000D 00-1.000D 00

PARAMETERS OF TIME FUNCTION

TIME OF RISING: TR = 5.0000D-03 TD = 1.0000D-02FO = 2.4000D 03 TIME OF DECAY: MAXIMUM VALUE:

SYMMETRICAL STRUCTURE AND LOADING SOLUTION FOR HALF STRUCTURE STATIC OUTPUT OPTION 2

AXIS OF STRUCTURE DESCRIBED BY A SMOOTH CURVE

STATIC SOLUTION REQUIRED: YES

NO KEEP OPTIONS EXERCIZED

ACCELERATION OF GRAVITY

SELF WEIGHT ADDED TO STATIC LOADS

NUMBER OF DYNAMIC LOADINGS

DYNAMIC OUTPUT OPTION

OUTPUT INTERVAL

TIME LIMIT

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 Dynarch application problems - two-hinged circular arch of reinforced concrete

PROBLEM DAP1

CIRCULAR ARCH - 180 DEG. - SYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

TABLE 7. - COLLAPSE PARAMETERS

DISPLACEMENT LIMITS

MAXIMUM HORIZONTAL DISPL: 3.0000D 00 MAXIMUM VERTICAL DISPL: 4.0000D 00

SHEAR	LIMITS		,
	TERM		SHEAR
	STA.		VALUE

13 3.00000 05

INTERACTION DIAGRAM DATA

T ERM	AXIAL FORCE	MOMENT
STA.	Multiplier	MULTIPLIER

13 1.0000D 05 1.0000D 05

AXIAL FORCE INPUT VALUES 0.0 0.800 1.800 3.200 4.720 3.200 1.800 0.800 0.0

MOMENT INPUT VALUES

-4.500-10.500-16.500-12.000 0.0 12.000 16.500 10.500 4.500

TABLE 8. - STATION COORDINATES

STA.	X-COORD	Y-COORD
1	0.0	0.0
2	1.5143D 00	2.3104D 01
3	6.0313D 00	4.5812D 01
4	1.3474D 01	6.7736D 01
5	2.3714D 01	8.8501D 01
6	3.6577D 01	1.0775D 02
7	5.1843D 01	1.2516D 02
8	6.9250D 01	1.4042D 02
9	8.8501D 01	1.5329D 02
10	1.09270 02	1.6353D 02
11	1.3119D 02	1.7097D 02
12	1.5390D 02	1.7549D 02
13	1.7700D 02	1.7700D 02

.

COMPLETE	RESPONSE,	TIME = 0.0				
STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0•0		2 -2 37400 03	6.0000D 00
2	-6.8170D-03	3.0856D-04	-1.4378D 04	-3 (5310 0	2 - 2 23620 03	6.0000D 00
3	-1.2130D-02	1.23280-03	-2.23730 04	-1 15550 0	2 - 2 • 2 3 0 2 0 3	6.0000D 00
4	-1.5086D-02	2.1092D-03	-2.5048D 04	4 41 200 0	2 - 2.07049 03	6.0000D 00
5	-1.5549D-02	2.2157D-03	-2.3517D 04	0.01290 0	1 -1.66620 03	6.0000D 00
6	-1.3933D-02	1.0171D-03	-1.8903D 04	1.99280 0	2 -1.69310 03	6.0000D 00
7	-1.0978D-02	-1.6904D-03	-1.2303D 04	2.85070 0	2 - 1.50040 03	6.0000D 00
8	-7.5325D-03	-5.7350D-03	-4.7496D 03	3.28250 0	2 -1.31680 03	6.0000D 00
9	-4.3531D-03	-1.0614D-02	2.8212D 03	3.27000 0	2 -1.15050 03	6.0000D 00
10	-1.9582D-03	-1.5603D-02	9.6000D 03	2.92/90 0	2 -1.00840 03	6.0000D 00
11	-5.5213D-04	-1.9907D-02	1.4930D 04	2.30190 0	2 - 8.96500 02	6.0000D 00
12	-2.1935D-05	-2.2816D-02	1.8325D 04	1,4667D 0	2 -8.1929D 02	6.0000D 00
13	0.0	-2.3844D-02	1.9491D 04	5.0330D 0	1 -7.7989D 02	6.0000D 00

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH APPLICATION PROBLEMS - TWO-HINGED CIRCULAR ARCH OF REINFORCED CONCRETE

CIRCULAR ARCH - 180 DEG. - SYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

COMPLETE RESPONSE, TIME = 0.0

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0	-6 20990	02 -2 22820 02	6.0000D 00
2	-6.8170D-03	3.08560-04	-1.4385D 04	-3 (5310		6.0000D 00
3	-1.2130D-C2	1.23280-03	-2.2383D 04	-3.49510		6.0000D 00
4	-1.5086D-02	2.1092D-03	-2.5060D 04	- 1. 15550	02 -2.06170 03	6.0000D 00
5	-1.5549D-02	2.2157D-03	-2.3528D 04	6+61290	01 -1.87980 03	6.0000D 00
6	-1.3933D-02	1.01710-03	-1.8912D 04	1.99280	02 -1.68950 03	6.0000D 00
7	-1.0978D-02	-1.6904D-03	-1.2309D 04	2.85070	02 - 1. 50000 03	6.0000D 00
8	-7.53250-03	-5.7350D-03	-4.7519D 03	3.26250	02 -1.31970 03	6.0000D 00
9	-4.3531 D-03	-1.0614D-02	2.8225D 03	3.27000	02 -1.15640 03	6.0000D 00
10	-1.9582D-03	-1.56030-02	9.6045D 03	2.9279D	02 -1.0170D 03	6.0000D 00
11	-5.5213D-04	-1.9907D-02	1.4937D 04	2.3019D	02 -9:07280 02	6.0000D 00
12	-2.1935D-05	-2.2816D-02	1.8334D 04	1•4667D	02 -8.3160D 02	6.0000D 00
13	0.0	-2.3844D-C2	1.9500D 04	5.03300	01 -7.92990 02	6.0000D 00

.

.

PROBLEM DAP1

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH APPLICATION PROBLEMS - TWO-HINGED CIRCULAR ARCH OF REINFORCED CONCRETE

PROBLEM DAP1

CIRCULAR ARCH - 180 DEG. - SYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

COMPLETE RESPONSE, TIME = 2.8205D-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0	-6 44490	02 -7 25490 03	6.0000D 00
2	-6.9837D-03	4.41370-05	-1.4922D 04	- 0. 44 470	02 -7.23470 03	6.0000D 00
3	-1.24860-02	6.81170-04	-2.2014D 04	-3.00350	02 -7.00240 03	6.0000D 00
4	-1.5760D-02	1.2891D-03	-2.4702D 04	-1.16080	02 - 8.56340 03	6.0000D 00
5	-1.6712D-02	1.1008D-03	-2.3310D 04	6+01260	01 -9.87175 03	6.0000D 00
6	-1.5726D-02	-3.2656D-04	-1.5605D 04	3.32810	02 -1.1489D 04	6.0000D 00
7	-1.3871D-02	-3.02100-03	-1.5656D 03	6.0540D	02 -1.3389D 04	6.0000D 00
 8	-1.2398D-C2	-6.01670-03	-2.8284D 03	-5.4546D	01 -1.5679D 04	6.0000D 00
9	-1.1424D-02	-9.4643D-03	-5.1605D 04	-2.1068D	03 -1.8686D 04	6.0000D 00
10	-8-65820-03	-1.79340-02	-1.17710 05	-2.8552D	03 -2.2538D 04	6.0000D 00
11	-3.33850-03	- 3- 86350-02	-7.50980 04	1.8404D	03 -2.6002D 04	6.00000.00
12	9.04130-04	-4 75000-02	1 33300 05	9.00170	03 -2.80170 04	6 00000 00
12	0.04150-04	-0.19900-02	1.55500 05	1.8841D	03 -2.7293D 04	
13	0.0	-8.24030-02	T+10A5D 02			5.12570 00

COMPLETE RESPONSE, TIME = 5.9583D-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0			6.0000D 00
2	-2.6369D-02	-1.1010D-03	-1.0604D 0	-4.58040 5	03 -5.16780 04	6.0000D 00
3	-4.2136D-02	-1.62650-03	-8.9152D 0	7.2945D 4	02 -5.1443D 04	6.0000D 00
4.	-4.8670D-02	-1.49880-03	4.1126D 0	4.0285D 3	03 - 5.0385) 04	6.0000D 00
5	-5,69100-02	-2.08540-03	7.59580 0	3.1036D	03 -5.0334D 04	6.0000D 00
-	-7 07330-02	5 48420-03	2.29850 0	-2.28.80D	03 -5.13710 04	6.00000.00
	-1.01350-02	1 20820-03	-1 22200 0	-6.2802D	03 -5.5213D 04	4 000000 00
	-8.80300-02	1.39820-02	-1.22390 0	- 4.3311D	03 -5.9388D 04	6.00000 00
. 8	-9.2994D-02	1.74600-02	-2.22660 0	5 -1.5974D	03 -6.2534D 04	6.00000 00
9	-8.8172D-02	-4.33800-04	-2.5963D 0	5 -4.1351D	03 -6.6033D 04	6.5339D 00
10	-6.64250-02	-4.91260-02	-3.5536D 0	5	04 -7.3652D 04	8.0323D 00
11	- 2.8883D- 02	-1.98060-01	-6.9836D 0	4	04 -7.20320 04	6.0000D 00
12	4.63160-03	-3.54680-01	2.5515D 0	5	04 -7 41850 04	5.1257D 00
13	0.0	-4.72570-01	5.3191D 0	5	UT = 1 + 1 0 JU UT	3.6922D 00

•

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHDMA, APRIL 1974 Dynarch Application problems - Two-Hinged Circular Arch of Reinforced Concrete

PROBLEM DAP1

CIRCULAR ARCH - 180 DEG. - SYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

COMPLETE RESPONSE, TIME = 8.8356D-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
. 1	0.0	0.0	0.0	3 10120		6.0000D 00
2	-6.5191D-02	2.2423D-03	7.3648D 03	3.10120	02 -2.78100 04	6.0000D 00
3	-1.3032D-01	1•4431D-02	-7.0143D 04	- 3. 34 /80	03 - 2. 76035 04	6.0000D 00
4	-1.8728D-01	3.0341D-02	-1.5694D 05	-3.74950	03 -2.8067D 04	6.0000D 00
5	-2.2514D-01	4.9484D-02	-1.1893D 05	1.6419D	03 -2.7788D 04	6.0000D 00
6	- 2. 52 38D-01	6.2586D-02	-5.5959D 04	2.7201D	03 - 2.75820 04	6.0000D 00
7	-2.6949D-01	7.96 56D-02	-7.8579D 04	-9.8126D	02 -2.80330 04	6.0000D 00
8	-2-84070-01	8-80400-02	+ 2,24510 05	~6.3002D	03 -3.4802D 04	7.75680.00
	-2 62960-01	5 49450-02	-4 93150 05	-1.1171D	04 -4.1823D 04	P 45200 00
9	-2.02900-01	5.47450-02	-4.03100 00	2.5631D	03 -4.55620 04	8.49200 00
10	-1.82220-01	-1.41220-01	-4.23840 05	2.0658D	04 -4.2576D 04	8.32100 00
11	-7.1718D-02	-4.6999D-01	5.44750 04	1.0523D	04 -3.6825D 04	6.0000D 00
12	-1.3960D-02	-8.0578D-01	2.9803D 05	7.3854D	03 - 3.4698D 04	3.80110 00
13	0.0	-1.05570 00	4.6901D 05			3.2112D 00

COMPLETE RESPONSE, TIME = 8.9298D-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0			6.0000D 00
2	-6.70730-02	2 .53350- 03	1.1481D 04	4.95930 0	2 -2.56330 04	6.0000D 00
3	-1.3449D-01	1.5158D-02	-7.3728D 04	-3.6804D 0	3 -2.6165D 04	6.0000D 00
4	-1-93220-01	3-18610-02	-1.6344D 0	-3.8752D 0	3 -2.5783D 04	6.0000D 00
5	- 2 32090-01	5 16900-02	-1:21080 0	1.8296D 0	3 -2.56760 04	6.00000.00
		4 52000 02		2.66150 0	3 - 2.6007D 04	
0	-2.60160-01	0.02990-02	-2.94000 04	-6.7258D 0	2 -2.65650 04	8.00000 00
7	-2.7753D-01	8.2953D-02	-7.5039D 04	-6.7041D 0	3 -3.3706D 04	6.0000D 00
8	-2.9286D-01	9.19540-02	-2.3022D 0	5 -1.0942D 0	4 -4.10470 04	7.9138D 00
9	-2.69530-01	5.5616D-02	-4.8354D 0	5 2,8519D 0	3 - 4. 37420 04	8.45200 00
10	-1.8598D-01	-1.46030-01	-4.1754D 0	5		8.3210D 00
11	-7.36000-02	-4.7879D-01	4.3842D 0	4		6.0000D 00
12	-1.51240-02	-8.2005D-01	2.9027D_0	1.06470 0 5	4 -3.31240 04	3.6766D 00
13	0.0	-1.07520 00	5.01630 0	9.1292D 0	3 -4.4022D 04	3.4362D 00

PROGRAM DYNARCH ANALYSIS AND PREDICTION OF COLLAPSE OF PLANE ARCHES UNDER STATIC AND DYNAMIC LOADING JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 DYNARCH APPLICATION PROBLEMS - TWO-HINGED CIRCULAR ARCH OF REINFORCED CONCRETE PROBLEM DAP2 CIRC. ARCH - 180 DEG. - NONSYN. DYN. LOAD AND SELF WEIGHT - INELASTIC TABLE 3. - STRESS-STRAIN CURVES CURVE NO. 1 MATERIAL SPECIFIC WEIGHT 8.694D-02 COMPRESSIVE ULTIMATE STRAIN 3.000D-03 STRESS VALUE SCALE FACTOR 1.000D 03 STRAIN VALUE SCALE FACTOR - 1.000D-03 STRESS INPUT VALUES -1.500 -3.900 -4.000 -3.800 -2.880 0.000 0.000 0.000 0.000 0.000 STRAIN INPUT VALUES -7.200 -2.400 -1.900 -1.350 -0.800 20.000 21.000 22.000 23.000 24.000 CURVE NO. 2 MATERIAL SPECIFIC WEIGHT 2.8610-01 COMPRESSIVE ULTIMATE STRAIN 2.0000-02 STRESS VALUE SCALE FACTOR 1.000D 04 STRAIN VALUE SCALE FACTOR 1.000D-03 STRESS INPUT VALUES -4.840 -4.830 -4.820 -4.810 -4.800 4.800 4.810 4.820 4.830 4.840 STRAIN INPUT VALUES -19.600-15.100-10.600 -6.100 -1.600 1.600 6.100 10.600 15.100 19.600 TABLE 5. - STATIC LOADS INTERMEDIATE INITIAL FINAL Y STA -STA. ST A. х X Y х Y -1.000D-15 0 0.0 0.0 0 0.0 0.0 1 0.0 25 0.0 -1.000D-15 0 0.0 0.0 0 0.0 0.0

TABLE 6. - DYNAMIC LOADING

DYNAMIC LOADING NO. 1

INITIAL				INTERMEDIATE			FINAL		
STA.	X	Y	STA.	x	Y	STA.	x	Ŷ	
5	1.000D-15-1.	,000p-15	6	7.0710-01-7	.0710-01	7	1.0000 00-	-1.000D 00	
7	1.0000 00-1.	.000D 00	8	7.071D-01-7	• 071D-01	9	1.000D-15-	-1.000D-15	

PARAMETERS OF TIME FUNCTION

TIME	OF	RISING:	TR	=	5.0000D-	03
TIME	0F	DECAY:	TD	Ŧ	1.0000D-	02
MAXIM	UM	VALUE:	FO	×	4.0000D	03

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 Dynarch application problems - Two-Hinged Circular Arch of Reinforged concrete

PROBLEM DAP2

CIRC. ARCH - 180 DEG. - NONSYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

COMPLETE RESPONSE, TIME = 3.7969D-03

STA.	X DISPL	Y DISPL	MOMENT	SHEAR	THRUST	CENTROID
1	0.0	0.0	0.0	- 9 51220	03 - 9 50100 0/	6.0000D 00
2	-3.1996D-02	-2.60370-03	-1.9707D 0	5	03 - 8. 39100 04	6.0000D 00
3	-4.4321D-02	-6.23570-03	-3.1823D 0	-5.2346U	03 -8.83130 04	6.0000D 00
4	-2.4685D-02	-1.7073D-02	-3.2830D 0	-4.3465D	02 -9.01900 04	6.5339D 00
5	3.17250-02	-5.4183D-02	-2.2996D 0	4.2487D 5	03 -9.3792D 04	6.0000D 00
6	1.0588D-01	-1.0696D-01	1.7215D 0	1•7369D	04 -9.4834D 04	6.0000D 00
7	1.53920-01	-1.6094D-01	4.3811D 0	1.1492D	04 -9.01760 04	4.0113D 00
8	1.07560-01	-1.21390-01	1.8202D 0	-1.1065D	04 -8.63900 04	6.0000D 00
9	6-12560-02	-5, 54 5 1 D-02	-2.0446D 0	-1.6694D	04 -8.41450 04	6.00000 00
10	2 83910-02	-5.71510-03	-2.55530 0	- 2.2063D	03 -7.4554D 04	6.92660.00
11	2.08930-02	0.02440-02	-2 24910 0	1.2364D	03 -6.3888D 04	4 00000 00
11	2.08730-02	7.02400-03	- 5 20550 0	7.4715D	03 -5.4516D 04	6.00000 00
12	1.81610-02	1.10080-04	- 3. 39350 04	4.9004D	03 -4.55410 04	6.00000 00
13	1.72930-02	-1.24340-02	5.94940 04	+ -1.4860D	02 -3.90230 04	6.0000U 00
14	1.4037D-02	-1.86570-02	5.6054D 04	4 -1.4342D	03 -3.49850 04	6.0000D 00
15	1.23760-02	-1,8480D-02	2.28520 0	4 -6.1217D	02 - 3.02420 04	6.0000D 00
16	1.1322D-02	-1.54810-02	8.6799D 03	3 -1.9929D	02 -2.68310 04	6.0000D 00
17	1.1632D-02	-1.15950-02	4.0660D 0	-2.8083D	02 -2.3001D 04	6.0000D 00
18	1.28210-02	-7.3104D-03	-2.43550 0	3 -3,4903D	$02 - 1_{2}9984D 04$	6.0000D 00
19	1.46920-02	-3.47600-03	-1.05160 04	+	02 - 1. 70640 04	6.0000D 00
20	1.63810-02	-6.50750-04	-1.7610D 04	~2 11600	02 -1 45240 04	6.0000D 00
21	1.7074D-02	8.0331D-04	-2.2509D 04	-2.11000 4	02 -1.45240 04	6.000 00 00
22	1• 5956D- 02	1.0526D-03	-2.4238D 04	-1.40010	01 -1.22870 04	6.0000D 00
23	1.25930-02	5.3634D-04	-2.1947D 04	9.8966D	01 -1.04550 04	6.0000D 00
24	7.0391D-03	-2.93120-05	-1.5153D 04	2.9344D	02 -9.1988D 03	6.0000D 00
25	0.0	0.0	0.0	6.5447D	02 -8.59100 03	6.0000D 00

JERSON DUARTE GUIMARAES - OSU - STILLWATER, OKLAHOMA, APRIL 1974 Dynarch Application problems - Two-Hinged Circular Arch of Reinforced Concrete

CIRC. ARCH - 180 DEG. - NONSYM. DYN. LOAD AND SELF WEIGHT - INELASTIC

7

NUNMAL IENMINATION OF CALCOTIC	NORMAL	TERMINAT1	ION OF	EXECUTION
--------------------------------	--------	-----------	--------	-----------

STA.	X DISPL	Y DISPL	MOMENT		SHEAR		THRUST		CENTROIC	כ
1	0.0	0.0	0.0		-1 41440	04	-1 22550	0E	6.0000D	00
2	-7.4027D-02	-1.49980-03	-3.7374D (05	-1.01400	04	-1.22550	05	6.0000D	00
3	-1.1003D-01	-4.39160-03	-3.8524D (05	-4.9/210	02	-1.23390	05	7.5499D	00
4	-5.4768D-02	-3.32470-02	-4.1856D (05	-1.43960	03	- 1.23440	05	7.5274D	00
5	9•9060D-02	-1.29940-01	-2.2457D (05	8•3854D	03	-1.31190	05	6.0000D	00
6	2.70150-01	-2.41150-01	3.8205D (05	2.6196D	04	-1.31530	05	6.0000D	00
7	3.7848D-01	-3.62940-01	5.6022D (05	7.7020D	03	-1.26450	05	3.8494D	00
8	2.6716D-01	-2.72300-01	3.9727D (05	-7.04530	03	-1.2186D	05	6.0000D	00
9	1.70590-01	-1.13830-01	-2.1639D (05	- 2. 6495D	04	-1.2098D	05	6.00000	00
10	7.87110-02	2.23610-02	-4.14770 (05	-8.5766D	03	-1.09790	05	7-86440	00
11	6 54 220-02	5 70390-02		05	3.7209D	03	-9.7023D	04	6.12970	00
** 12	5.44040-02	5 54500-02	-2 10200 (05	5.1134D	03	- 8. 6566D	04	4 00000	00
12	5.00040-02	3.34300-02	-2.10200	~	6.7769D	03	-7.73410	04	6.00000	00
13	5.69890-02	3.30070-02	-5.33900 0	04	6.5510D	03	-6.9179D	04	6.00000	00
14	4.82610-02	7.04590-03	9.8236D (04	4.1488D	02	-6.3660D	04	6.0000D	00
15	4.4014D-02	- E. 4676D-03	1.0784D (05	-3.15110	03	-5.77020	04	6.0000D	00
16	3•7434D-02	-1.05210-02	,3 . 4901D (04	-1.6611 D	03	-5.2806D	04	6.0000D	00
17	3.64590-02	-9.93310-03	-3.55530 (03	-4.9606D	01	-4.7811D	04	6.0000D	00
18	3.2804D-02	-7.42370-03	-4.7036D (03	-5.30150	00	- 4. 33930	04	6.0000D	00
19	3.1906D-C2	-6.7633D-03	-4.82630 (03	- 3- 01930	02	-4.06270	04	6.0000D	00
20	2.94110-02	-4.3894D-03	-1.18160 (04	-2-43320	02	-3-91790	04	6.00000	00
. 21	2.7565D-02	-3.85190-03	-1.7449D (04	-1.41630	02	-3-79200	04	6.0000D	00
22	2.3850D-02	-2.63620-03	-2.0728D	04	-2 55950	02	-3 81540	04	6.0000D	00
23	1.8835D-02	-2.35620-03	-2.8966D (04	-3.41240	02	-2 70770	0 4	6.0000D	.00
24	1.1099D-02	-1.46450-03	-3.6868D (04		02	- 2. 19(10	04	6.0000D	00
25	0.0	0.0	0.0		1.04700	05	-2.042/0	U 4	6.00000	00

FAILURE OCCURRED AT TIME = 5.18780-03 SECS.

FAILURE DUE TO CRUSHING OF TOP FIBERS AT JOINT

COMPLETE RESPONSE, TIME = 5.18780-03

PROBLEM DAP2

TABLE D.1

NONDIMENSIONAL RESULTS--PROBLEM DTP2.1

Elapsed Time	Vertical Displ. Crown	Thrust Crown	Moment ⅓ Point	Moment Crown	Moment Crown
t/T _o	^V dc p _o R ² ∕AE	$\frac{T_{c}}{P_{0}R}$	M ₁₄ p ₀ Rr	M _c p _o Rr	$\frac{EI}{p_0 R^3 r} \left[\frac{\partial^2 u_r}{\partial \theta^2} + u_r \right]_c$
0.1	-0.1906	-0.1896	-0.0060	-0.0026	-0.0026
0.2	-0.6920	-0.6335	-0.0257	-0.0097	-0.0097
0.3	-1.3310	-1.2002	0.0386	-0.0138	-0.0138
0.4	-1.8783	-1.6766	0.2711	-0.0321	-0.0321
0.5	-2.1276	-1.8904	0.4879	-0.1634	-0.1629
0.6	-2.0428	-1.8139	0.5038	-0.4226	-0.4213
0.7	-1.8336	-1.4362	0.3847	-0.4964	-0.4950
0.8	-1.7526	-0.9484	0.1798	-0.1139	-0.1136
0.9	-1.8304	-0.4858	-0.1773	0.4699	0.4685
1.0	-1.9881	-2.2144	-0.6113	0.8927	0.8900
1.1	-2.1864	-0.2288	-0.8684	1.1000	1.0967
1.2	-2.3454	-0.4691	-0.7592	1.0333	1.0302
1.3	-2.3705	-0.8968	-0.3277	0.6202	0.6183
1.4	-2.2546	-1.3319	0.2122	0.0354	0.0353
1.5	-2.0495	-1.6812	0.6181	-0.5019	-0.5005
1.6	-1.8269	-1.8201	0.7265	-0.8076	-0.8053
1.7	-1.6753	-1.6818	0.5628	-0.6402	-0.6384
1.8	-1.5604	-1.3444	0.2728	-0.0760	-0.0758
1.9	-1.3132	-0.8560	-0.0863	0.4040	0.4028
2.0	-0.8951	-0.4166	-0.4201	0.5179	0.5163
2.1	-0.4716	-0.1361	-0.4572	0.3861	0.3850
2.2	-0.2312	-0.1249	-0.1416	0.1502	0.1498
2.3	-0.2558	-0.4168	0.2271	-0.1740	-0.1735
2.4	-0.5533	-0.8927	0.4494	-0.4978	-0.4963
2.5	-1.0829	-1.4452	0.5339	-0.6154	-0.6136
2.6	-1.6854	-1.8219	0.4858	-0.4546	-0.4533
2.7	-2.1279	-1.9152	0.3603	-0.1621	-0.1617
2.8	-2.2456	-1.6823	0.2264	0.0968	0.0964
2.9	-2.0355	-1.1896	0.0358	0.2148	0.2141
3.0	-1.6813	-0.6516	-0.1931	0.2053	0.2047

.

vita $^{\sim}$

Jerson Duarte Guimarães

Candidate for the Degree of

Doctor of Philosophy

Thesis: A METHOD OF ANALYSIS FOR NONLINEAR DYNAMIC RESPONSE OF ARCHES

Major Field: Civil Engineering

Biographical:

- Personal Data: Born in Pitangui, State of Minas Gerais, Brazil, on January 25, 1923, son of Joao Antonio Guimaraes and Izaura Maria Duarte.
- Education: Graduated from Colegio Estadual de Minas Gerais, Belo Horizonte, Brazil, in February, 1944, junior high school; graduated from Colegio Anchieta de Belo Horizonte, Minas Gerais, Brazil, in December, 1946, high school; graduated accountant from Escola Technica de Comercio Brasileira, Belo Horizonte, Brazil, in December, 1944; received Civil Engineering degree from Escola de Engenharia da Universidade Federal de Minas Gerais, Belo Horizonte, Brazil, in December, 1952; received Master of Science degree in Civil Engineering from Duke University, Durham, North Carolina, USA, in June, 1967.
- Professional Experience: Worked for Mrs. A. J. Diniz & Cia., auto dealers, Belo Horizonte, Minas Gerais, Brazil, from 1941 to 1952, first as an auxiliary bookkeeper and then, from 1945, as Chief Accountant; Assistant Civil Engineer for Sociedade Brasileira de Eletrificação and for Construtora Walter Coscarelli, Belo Horizonte, Minas Gerais, Brazil, from 1953 to 1955; Consultant Structural Engineer and Professor of Civil Engineering in Goiania, State of Goias, Brazil, since 1957; Executive Director of the Escola de Engenharia da Universidade Federal de Goias, Goiania, Brazil, from 1959 to 1962; graduate teaching assistant at Duke University, Durham, North Carolina, from September, 1965 to March, 1967, and at Oklahoma State University for the academic year of 1973-1974.
- Honorary: Vice-President for the Clube de Engenharia de Goias, Goiania, Goias, Brazil, 1959/60; member of the first University Council of the Universidade Federal de Goias, representing the

Escola de Engenharia, Goiania, Goias, Brazil, 1961; Vice-President of the Goias State branch of the Conselho Federal de Engenharia, Arquitetura e Agronomia, Goiania, Goias, Brazil, 1968.