## BEAM CONSTANTS BY THE STRING POLYGON METHOD Cin: 6 : 160

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The purpose of the thesis is the derivation of the general formulas for beam constants for beams with parabolic haunches. The string polygon method is used for the derivation of these formulas, and the evaluation of the complicated integration function is made by means of Ritter's approximation. Tables of beam constants for the most important cases are included in this thesis.
The string polygon method mentioned above was presented by Professor J. J. Tuma in his course CE-620-Ph.D. Seminar in the Spring of 1959.

## CHAPTER I

## THE STRING POLYGON EQUATION

A simple beam of variable section loaded by a general system of - forces is considered (Fig. l-1).

(B)

Fig. 1-1

The elastic curve of this beam is shown in an exaggerated form. Three arbitrarily selected points of the elastic curve are denoted as i, $j$, and $k$. The change in slope between the line $i j$ and $j k$ is designated by $\emptyset_{j}$, (the change in slope of the string polygon). The algebraic expression for this change in slope is given by the equation (1-1)

$$
\begin{equation*}
\varphi_{j}=G_{i j} \mathbb{M}_{i}+\left(F_{j i}+F_{j k}\right) M_{j}+G_{k j} M_{k}+\tau_{j i}+\tau_{j k} \tag{1-1}
\end{equation*}
$$

The notation of this equation follows:
$M_{i}=$ the bending moment of the simple beam at $i$.
$M_{j}=$ the bending moment of the simple beam at $f$.

$$
\begin{aligned}
& \mathrm{M}_{\mathrm{k}}=\text { the bending moment of the simple beam at } \mathrm{k} \text {. } \\
& \mathrm{F}_{\mathrm{ji}}=\text { the angular flexibility of the equivalent simple beam } \\
& \text { ij at } j \text {. } \\
& F_{j k}=\text { the angular flexibility of the equivalent simple beam } \\
& \text { jk at j. } \\
& G_{i j}=\text { the angular carry over value of the equivalent simple } \\
& \text { beam ij at } 1 . \\
& G_{k j}=\text { the angular carry over value of the equivalent simple } \\
& \text { beam } j k \text { at } k \text {. } \\
& \tau_{j 1}=\text { the angular load function of the equivalent simple beam } \\
& \text { ij at } j \text {. } \\
& \tau_{j k}=\text { the angular load function of the equivalent simple beam } \\
& j k \text { at } j \text {. } \\
& \text { It may be easily proved that the changes in slope of the string } \\
& \text { polygon } \varnothing_{i}, \varnothing_{j}, \varnothing_{k} \text {, when applied as elastic loads on the equivalent } \\
& \text { conjugate beam,develop shears and moments which, at given points } 1 \text {, } \\
& j \text {, } k \text {, are equal to the slopes and deflections of the real beam, re- } \\
& \text { spectively. A complete derivation of the equation (1-1) and the proof } \\
& \text { of the statement mentioned above can be found elsewhere (1). }
\end{aligned}
$$

## CHAPTER II

## RITTER'S FORMULA

A prismatic member of parabolic variation is considered (Fig. 2-1). The depths of the beam at the endsis denoted by $h_{0}$ and $h_{B}$ respectively. The length of this member is L $\alpha$. The depth of the arbitrarilyselected section given by position coordinate x is ." ..

$$
\begin{equation*}
h_{x}=h_{0}+\frac{h_{0} \omega}{(I \alpha)^{2}} x^{2}=h_{0}\left(1+\frac{\omega}{(L \alpha)^{2}} x^{2}\right)=h_{0} t_{x} \tag{2-1}
\end{equation*}
$$

where $t_{x}=$ variable parameter.


Fig. 2-1

The moment of inertia with respect to principal axis z-z is

$$
\begin{equation*}
I_{z}=\frac{b h_{x}{ }^{3}}{12}=\frac{b h_{0}{ }^{3}}{12} t_{x}^{3}=I_{0} t_{x}^{3} \tag{2-2}
\end{equation*}
$$

where

$$
\begin{aligned}
& I_{0}=\text { the moment of inertia of the section at left end. } \\
& t_{x}=\text { the variable parameter defined by Eq. }(2-1) .
\end{aligned}
$$

In the analysis of these members tro typical integral expressions frequently occur:

and

$$
\int_{0}^{L \alpha k} \frac{x^{n} d x}{E I_{x}}=\frac{1}{E I_{0}} \int_{0}^{L \alpha k} \frac{x^{n_{d x}}}{t_{x}}=\frac{1}{E I_{0}}(L \alpha k)^{n+1} Q_{n}\left(L \alpha_{k}\right) \quad(2-4 a)
$$

As the evaluation of these functions is laborious and time consuming, many approximate formulas for the solution of these expressions have been proposed. The most powerful approach has been suggested by Ritter (2). The application of the Ritter's formula to the evaluation of the Q's function is shown in the following part of this thesis. With the notation

$$
\begin{equation*}
h_{B}=h_{0}+h_{0} \omega=h_{0} \gamma \tag{2-5}
\end{equation*}
$$

as show in Fig. (2-1), the general Q functions of Eqs. (2-3a) and
(2-4a) become:

$$
\begin{equation*}
\int_{0}^{L \alpha} \frac{x^{n} d x}{E I_{x}}=\frac{1}{E I_{0}} \int_{0}^{L \alpha} x^{n}\left(\frac{h_{0}}{h_{x}}\right)^{3} d x \tag{2-3b}
\end{equation*}
$$

and

$$
\begin{equation*}
\int_{0}^{I \alpha k} \frac{x^{n} d x}{E I_{x}}=\frac{1}{E I_{0}} \int_{0}^{L \alpha k} x^{n}\left(\frac{h_{0}}{h_{x}}\right)^{3} d x \tag{2-4b}
\end{equation*}
$$

If the function $\left(\frac{h_{0}}{h_{X}}\right)^{3}$ is assumed to be a parabola of $2 r$ degree, the following relationship can be stated:

$$
\begin{equation*}
\left(\frac{h_{0}}{h_{x}}\right)^{3}=1-c_{B}\left(\frac{x}{L}\right)^{2 r} \tag{2-6}
\end{equation*}
$$

The numerical constants $C_{B}$ and $r$ are unknown and must be computed from some special conditions. The graphical interpretation of this equation is shown in Fig. (2-2).


Fig. 2-2

The extreme values of this function are:

$$
\begin{array}{ll}
x=0, & \frac{h_{0}}{h_{x}}=1 \\
x=L, & \frac{h_{0}}{h_{B}}=\frac{1}{\gamma} \tag{2-8}
\end{array}
$$

For the evaluation of the constants $C_{B}$ and $r$, one additional condition is necessary. This condition may be selected arbitrarily. For example:

$$
\begin{equation*}
x=\frac{L}{2}, \quad h_{C}=h_{0}\left(1+\frac{\omega}{4}\right) \tag{2-9}
\end{equation*}
$$

The meaning of the symbol $h_{C}$ is explained by Fig. (2-3). The results of Eqs. (2-7, 8, 9) are substituted in the Eq. (2-5) and the constants $C_{B}, C_{C}$, and $r$ are obtained:


Fig. 2-3

$$
\begin{equation*}
c_{B}=1-\left(\frac{1}{1+\omega}\right)^{3} \tag{2-10}
\end{equation*}
$$

$$
\begin{equation*}
c_{C}=1-\left(\frac{1}{1+\frac{\omega}{4}}\right)^{3} \tag{2-11}
\end{equation*}
$$

$$
\begin{equation*}
\mathbf{r}=-1.66 \log \frac{\mathrm{C}_{C}}{\mathrm{C}_{B}} \tag{2-12}
\end{equation*}
$$

The relationship between $\omega$ and $r$ is computed by means of the Eqis. (2-10, 11, 12) and recorded in Table $I_{.}$.
table I.
THE RELATIONSHIP BETWEEN $\boldsymbol{\omega}$ AND $r$

| $\omega$ | $\left(\frac{1}{1+\omega}\right)^{3}$ | $C_{B}$ | $\frac{1}{\left(1+\frac{\omega}{4}\right)^{3}}$ | ${ }^{C} C$ | $\log \frac{C_{C}}{C_{B}}$ | r |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| 0.1 | . 751315 | . 248685 | . 9285576 | . 071424 | -. 541800 | . 899910 |
| 0.2 | . 578704 | . 421296 | . 863808 | . 136192 | -. 490394 | . 814527 |
| 0.3 | . 455166 | . 544834 | . 804992 | . 195008 | -. 446214 | . 741145 |
| 0.4 | . 364431 | . 635569 | . 751296 | . 248704 | -. 407479 | . 676808 |
| 0.5 | . 296296 | . 703704 | . 702336 | . 297664 | -. 373660 | . 620636 |
| 0.6 | . 244141 | . 755859 | . 657536 | . 342464 | -. 343825 | . 571081 |
| 0.7 | . 203542 | . 796458 | . 616448 | . 383552 | -. 317340 | . 527090 |
| 0.8 | . 171468 | . 528532 | . 578688 | . 421312 | -. 293701 | . 487827 |
| 0.9 | . 145794 | . 854206 | . 544000 | . 456000 | -. 272597 | . 452774 |
| 1.0 | . 125000 | . 875000 | . 512000 | . 488000 | -. 253592 | . 421207 |
| 1.1 | . 107980 | . 892020 | . 482496 | . 517504 | -. 236460 | . 392752 |
| 1.2 | . 093914 | . 906086 | . 455168 | . 544832 | -. 220909 | . 366922 |
| 1.3 | . 082190 | . 917810 | . 429888 | . 570112 | -. 206790 | . 343471 |
| 1.4 | . 072338 | . 927662 | . 406464 | . 593536 | -. 193942 | . 322131 |
| 1.5 | . 064000 | . 936000 | . 384704 | . 615296 | -. 182190 | . 302611 |
| 1.6 | . 056896 | . 943104 | . 364416 | . 635584 | -. 171385 | . 284664 |
| 1.7 | . 050805 | . 949195 | . 345600 | . 654400 | -. 161510 | . 268262 |
| 1.8 | . 045554 | . 954446 | . 328000 | . 672000 | -. 152384 | . 253104 |
| 1.9 | . 041002 | . 958998 | . 311680 | . 688320 | -. 144027 | . 239224 |
| 2.0 | . 037037 | . 962963 | . 296320 | . 703680 | -. 136237 | . 226285 |

## BASIC FUNCTIONS

The algebraic solution of the basic functions $Q^{\prime} s$ defined by Eqs. (2-3b,4b) for the most important cases is presented in this thesis.

The general solution of the Eq. (2-3b) follows

$$
\begin{align*}
Q_{n}^{(L \alpha)} & =\frac{1}{(L \alpha)^{n+1}} \int_{0}^{L \alpha} x^{n}\left(\frac{h_{0}}{h_{x}}\right)^{3} d x \\
& =\frac{1}{(L \alpha)^{n+1}} \int_{0}^{L \alpha} x^{n}\left[1-C_{B}\left(\frac{x}{L \alpha}\right)^{2 r}\right] d x \\
& =\frac{1}{(L \alpha)^{n+1}}\left[\frac{x^{n+1}}{n+1}-C_{B} \frac{x^{2 r+n+1}}{(L \alpha)^{2 r}(2 r+n+1)}\right] \\
& =\frac{1}{n+1}-\frac{C_{B}}{2 r+n+1} \tag{3-1}
\end{align*}
$$

If the exponent $n=0,1,2$, the $Q_{n}^{(L \alpha)}$ functions become:

$$
\left.\begin{array}{l}
Q_{0}^{(L \alpha)}=1-\frac{C_{B}}{2 r+1} \\
Q_{1}^{(L \alpha)}=2-\frac{C_{B}}{2 r+2}  \tag{3-2}\\
Q_{2}^{(L \alpha)}=3-\frac{C_{B}}{2 r+3}
\end{array}\right\}
$$

The general solution of the Eq. (2-4b) gives:

$$
\begin{align*}
Q_{n}^{(L \alpha k)} & =\frac{1}{(L \alpha k)^{n+1}} \int_{0}^{L \alpha k} x^{n}\left(\frac{h_{0}}{h_{x}}\right)^{3} 3^{2} d x \\
& =\frac{1}{(L \alpha k)^{n+1}} \int_{0}^{L \alpha k}\left[1-C_{B}\left(\frac{x}{L \alpha}\right)^{2 r}\right] x^{n d x} \\
& =\frac{1}{(L \alpha k)^{n+1}}\left[\frac{1}{n+1}-c_{B} \frac{x^{n+1}}{(L \alpha)^{2 r}(2 r+n+1)}\right]_{0}^{L \alpha k} \\
& =\frac{1}{(L \alpha k)^{n+1}}\left[\frac{(L \alpha k)^{n+1}}{n+1}-\frac{c_{B}}{(L \alpha)^{2 r}} \frac{(L \alpha k)^{2 r+n+1}}{2 r+n+1}\right] \\
& =\frac{1}{n+1}-\frac{c_{B} k^{2 r}}{2 r+n+1} \tag{3-3}
\end{align*}
$$

If the exponent $n=0,1,2$, the $Q_{n}{ }^{(L \alpha k)}$ functions become:

$$
\left.\begin{array}{l}
Q_{0}^{\left(L \alpha_{k}\right)}=  \tag{3-4}\\
1-\frac{C_{B} k^{2 r}}{2 r+1} \\
Q_{1}^{\left(L \alpha_{k}\right)}= \\
Q_{2} \stackrel{C_{B} k^{2 r}}{2 r+2} \\
Q_{2}^{\left(L \alpha_{k}\right)}= \\
3-\frac{C_{B} k^{2 r}}{2 r+3}
\end{array}\right\}
$$

The angular constants for a prismatic member with a parabolic haunch at the right end, simply supported at both ends are derived in a general form (Fig. 4-1).


Fig. 4-1
a) Angular Flexibilities and Carry Over Values. If the beam shown in Fig. (4-1) is acted upon by a unit moment applied at A, the end slopes of the elastic curve at $A$ and $B$ are called the angular flexibility $F_{A B}$ and the angular carry over value $G_{B A}$, respectively (Fig. 4-2). The bending moment, the string polygon, and the conjugate beam are shown in the same figure. The elastic weights $\varnothing_{A}, \varnothing_{D}, \phi_{B}$ shown in Fig. (4-2) and defined by Eq. (1-1) are:

$$
\begin{align*}
& \phi_{A}=F_{A D}+\beta G_{D A} \\
& \phi_{D}=G_{A D}+\beta\left(F_{D A}+F_{D B}\right)  \tag{4-1}\\
& \phi_{B}=\beta G_{D B} .
\end{align*}
$$



Elastic Curve


String Polygon


Bending Moment Diagram


Conjugate Beam

Fig. 4-2

The angular flexibility $\mathrm{F}_{\mathrm{AB}}$ is the left reaction of the conjugate beam (Fig. 4-2). From statics

$$
\begin{align*}
F_{A B} & =\phi_{A}+\phi_{D} \beta \\
& =\left(F_{A D}+2 \beta G_{A D}+\beta^{2} F_{D A}\right)+\beta^{2} F_{D B} \tag{4-2a}
\end{align*}
$$

or in terms of Q's (Eqs. 3-2)

$$
F_{A B}=\frac{L \alpha}{3 E I_{0}}\left(1+\beta+\beta^{2}\right)+\frac{L \beta 3}{E I_{0}}\left(Q \underset{0}{(L \beta)}-2 Q \underset{1}{(L \beta)}+Q_{Q}^{(L \beta)}\right) \cdot(4-2 b)
$$

The carry over value $G_{B A}$ is the right reaction of the conjugate beam (Fig. 4-2):

$$
\begin{align*}
G_{B A} & =\phi_{B}+\phi_{D} \alpha \\
& =\alpha\left(G_{A D}+\beta F_{D A}\right)+\beta\left(G_{D B}+\alpha F_{D B}\right) \tag{4-3a}
\end{align*}
$$

or In terms of Q's (Eqs. 3-2)

$$
G_{B A}=\frac{L \alpha^{2}}{6 E I_{0}}(1+2 \beta)+\frac{L \beta 2}{E I_{0}}\left[\alpha\left(Q \underset{0}{(L \beta)}-Q_{1}^{(L \beta)}\right)+\beta\left(Q\left(\underset{1}{(L \beta)}-Q \begin{array}{c}
(L \beta) \\
2
\end{array}\right] .\right.\right.
$$

If the beam shown in Fig. (4-1) is acted upon by a unit moment applied at $B$, the end slopes of the elastic curve at $A$ and $B$ are called the angular carry over value $G_{A B}$ and the angular flexibility $F_{B A}$, respectively (Fig. 4-3). The elastic weights $\phi_{A}, \varnothing_{D}$, $\phi_{B}$ in Fig. (4-3) are:


Elastic
Curve


String Polygon


Bending Moment Diagram


Conjugate
Beam

Fig. 4-3

$$
\left.\begin{array}{l}
\phi_{A}=\alpha G_{D A} \\
\phi_{D}=\alpha\left(F_{D A}+F_{D B}\right)+G_{B D} \\
\phi_{B}=\alpha G_{D B}+F_{B D}
\end{array}\right\}
$$

The angular carry over value $G_{A B}$ is the left reaction of the conjugate beam (Fig. 4-3).

$$
\begin{aligned}
G_{A B} & =\emptyset_{A}+\emptyset_{D} \beta \\
& =\alpha\left(G_{D A}+\beta F_{D A}\right)+\beta\left(G_{B D}+\alpha F_{D B}\right)
\end{aligned}
$$

or in terms of $Q^{\prime} s$

$$
\begin{array}{r}
G_{A B}=\frac{L \alpha^{2}}{6 E I_{0}}(1+2 \beta)+\frac{L \beta^{2}}{E I_{0}}\left[\alpha\left(Q{\underset{0}{(L \beta)}-Q(L \beta)}_{1}^{(L \beta}\right)+\beta(Q \underset{1}{(L \beta)}-Q \underset{2}{(L \beta)})\right] . \\
(4-5 b)
\end{array}
$$

The angular flexibility $F_{B A}$ is the right reaction of the conjugate beam (Fig. 4-3) :

$$
\begin{aligned}
F_{B A} & =\phi_{B}+\phi_{D} \alpha \\
& =\alpha^{2} F_{D A^{\prime}}+\left(F_{B D}+2 \alpha G_{B D}+\alpha^{2} F_{D B}\right)
\end{aligned}
$$

or in terms of $Q^{\prime} s$

$$
F_{B A}=\frac{L \alpha^{3}}{3 E I_{0}}+\frac{L \beta}{E I_{0}}\left(\alpha^{2} Q \frac{(L \beta)}{0}+2 \alpha \beta Q \frac{(L \beta)}{1}+\beta^{2} Q \frac{(\Gamma \beta)}{2}\right) . \quad(4-6 b)
$$

b) Angular Load Functions If the beam shown in Fig. (4-1) is
acted upon by a unit load moving gradually from $A$ to $B$, the end slopes of the elastic curve at $A$ and $B$ are called the influence values of angular load function. The general formulas of the influence values will depend upon the position of the unit load, that is, whether it is located within the haunch or in the straight part of the member.

Fig. ( $4-4$ ) shows a beam acted upon by a unit load applied in the straight part of the member. By method of superposition, the resultent effect can be obtained by adding together two partial effects (a) and (b). The angular load functions due to the bending moment of part (a) follow

$$
\left.\begin{array}{l}
\tau_{\mathrm{AB}}^{\prime}=\operatorname{Ln} \mathrm{F}_{\mathrm{AB}}  \tag{4-7}\\
\tau_{\mathrm{BA}}^{\prime}=\operatorname{LrG}_{\mathrm{BA}} .
\end{array}\right\}
$$

The elastic weights $\varnothing_{\mathrm{A}}^{\prime \prime}, \varnothing_{\mathrm{C}}^{\prime \prime}$ due to the bending moment of part (b) are:

$$
\left.\begin{array}{l}
\phi_{A}^{\prime \prime}=-\operatorname{Ln} F_{A C}  \tag{4-8}\\
\phi_{C}^{\prime \prime}=-\operatorname{LnG} \\
A C
\end{array}\right\}
$$

The angular load functions $\tau_{A B}^{\prime \prime}, \tau_{B A}^{\prime \prime}$ are the reactions of the conjugate beam at left and right ends, respectively.

$$
\left.\begin{array}{l}
\left.\tau_{A B}^{\prime \prime}=\phi_{A}+\phi_{C} n^{\prime}=-\operatorname{Ln}\left(F_{A C}+n^{\prime} G_{A C}\right)=-\frac{(L n}{6 E I_{0}}\right)^{2}\left(2+n^{\prime}\right) \\
\tau_{B A}^{\prime \prime}=\phi_{C} n=-L_{n}^{2} G_{A C}=-\frac{(L n)^{2}}{6 E I_{0}} . \tag{4-9}
\end{array}\right\}
$$



The angular load functions $\tau_{A B}, \tau_{B A}$ may be obtained by adding the Eqs. (4-7) and (4-9)

$$
\left.\begin{array}{l}
\tau_{A B}=\operatorname{Ln}\left[F_{A B}-\frac{L n}{6 E I_{0}}\left(2+n^{\prime}\right)\right] \\
\tau_{B A}=\operatorname{Ln}\left[G_{B A}-\frac{L n^{2}}{6 E I_{0}}\right] \tag{4-10}
\end{array}\right\}
$$

If the beam shown in Fig. (4-1) is acted upon by a unit load applied at a point within the haunch (Fig. 4-5), the angular load functions due to the bending moment of part (a) follow:

$$
\left.\begin{array}{l}
\tau_{\mathrm{AB}}^{\prime}=\operatorname{LnF}_{\mathrm{AB}}  \tag{4-11}\\
\tau_{\mathrm{BA}}^{\prime}=\operatorname{LnG}_{\mathrm{BA}} .
\end{array}\right\}
$$

The elastic weights $\varnothing_{A}^{\prime \prime}, \varnothing_{D}^{\prime \prime}, \varnothing_{C}^{\prime \prime}$ shown in the Fig. $(4-5 b)$ are:

$$
\left.\begin{array}{l}
\phi_{A}^{\prime \prime}=-\operatorname{Ln} F_{A D}-L k G_{D A} \\
\phi_{D}^{\prime \prime}=-\operatorname{Ln} G_{A D}-\operatorname{Lk}\left(F_{D A}+F_{D C}\right) \\
\phi_{C}^{\prime \prime}=-\operatorname{Lk} G_{D C}
\end{array}\right\}
$$

The angular load functions $\tau_{A B}^{\prime \prime}, \tau_{B A}^{\prime \prime}$ are the reactions of the conjugate beam at left and right ends, respectively.

$$
\begin{align*}
\tau_{A B}^{\prime \prime} & =\emptyset_{A}^{\prime \prime}+\beta \phi_{D}^{\prime \prime}+\phi_{C}^{\prime \prime} \\
& =-\frac{L^{2} \alpha}{6 E I_{0}}\left(6 n-3 n \alpha-3 \alpha+2 \alpha^{2}\right)-L k \beta^{2} F_{D C}-L k \beta n^{\prime} G_{D C} \tag{4-13a}
\end{align*}
$$



$$
\begin{align*}
\tau_{B A}^{n} & =\alpha \phi_{D}^{\prime \prime}+n \phi_{C}^{\prime \prime}  \tag{4-13a}\\
& =-\frac{(L \alpha)^{2}}{6 E I_{0}}(3 n-2 \alpha)-L k \beta \alpha F_{D C}-L k \beta n G_{D C}
\end{align*}
$$

or in terms of $Q s$

$$
\begin{aligned}
& \tau_{A B}^{\prime \prime}=-\frac{L^{2} \alpha}{6 E I_{0}}\left(6 n-3 n \alpha-3 \alpha+2 \alpha^{2}\right)-\frac{(L k \beta)^{2 \beta}}{E I_{0}}\left[\begin{array}{cc}
(L \beta k) \\
0 & -(k+1) \\
Q_{1}\left(\beta_{k}\right)_{k} & (L \beta k) \\
2
\end{array}\right] \\
& \tau_{B A}^{\prime \prime}=-\frac{(L \alpha)^{2}}{6 E I_{0}}(3 n-2 \alpha)-\frac{(L k \beta)^{2}}{E I_{0}}\left[\begin{array}{c}
\alpha Q_{0}^{(L \beta k)}+(k \beta-\alpha) \\
\underset{0}{(L \beta-13 b)} \underset{1}{(L \beta k)} \underset{2}{(L \beta k)}]
\end{array}\right]
\end{aligned}
$$

The angular load functions $\tau_{A B}, \tau_{B A}$ may be obtained by adding the Eqs. (4-12) and (4-13b) :

$$
\begin{aligned}
& \tau_{A B}=\operatorname{Ln}_{A B}-\left\{\frac{L^{2} \alpha}{6 E I_{0}}\left(6 n-3 n \alpha-3 \alpha+2 \alpha^{2}\right)+\frac{(L k \beta)^{2} \beta}{E I_{0}}\left[Q_{0}^{(I \beta k)}-(k+1) Q_{1}^{(L \beta k)}+k Q\left(L_{2} \beta k\right)\right]\right\} \\
& \left.\tau_{B A}=\operatorname{Ln} C_{B A}-\left\{\frac{(L \alpha)^{2}}{6 E I_{0}}(3 n-2 \alpha)+\frac{(L k \beta)^{2}}{E I_{0}}\left[\begin{array}{c}
(I B k) \\
0
\end{array}+(k \beta-\alpha) \mathcal{Q}_{1}^{(I \beta k)}-k \beta Q_{2}^{(L \beta k)}\right]\right\}\right] \\
& \text { (4-14) }
\end{aligned}
$$

The table of beam constants for the beam with one parabolic haunch is computed and listed in Tables II-and III. -
table II
BEAM CONSTANTS FOR BEAM WITH ONE PARABOLIC HAUNCH (END A)


| Right Haunch |  | $\begin{array}{c\|} \text { Angular } \\ \text { Flexi- } \\ \text { bilities } \end{array}$ | Angula Carry Over | Influence Values of Angular Load Functions; Coef. $\times \frac{\mathrm{PL}}{} \mathrm{EI}_{0}$ |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  |  | Value | n |  |  |  |  |  |  |  |  |
|  |  | f.x $\frac{1}{E I}$ | Coef. $\times$ | 0.1 | 0.2 | 0.3 | 0.4 | 0.5 | 0.6 | 0.7 | 0.8 | 0.9 |
| $\beta$ | $\omega$ | $F_{A B}$ | $\mathrm{G}_{\mathrm{AB}}$ | $\tau_{\text {AB }}$ | $\tau_{\text {AB }}$ | $\tau_{A B}$ | $\tau_{A B}$ | $\tau_{\mathrm{AB}}$ | $\tau_{\mathrm{AB}}$ | $\tau_{A B}$ | $\tau_{A B}$ | $\tau_{\mathrm{AB}}$ |
| 0:3 | 1.0 | . 3310 | . 1540 | . 0283 | . 0475 | . 0588 | . 0731 | . 0613 | . 0546 | . 0539 | . 0303 | . 0155 |
|  | 1.2 | . 3305 | . 1523 | . 0283 | . 0474 | . 0587 | . 0722 | . 0611 | . 0543 | . 0533 | . 0300 | . 0152 |
|  | 1.4 | . 3301 | . 1507 | . 0282 | . 0473 | . 0585 | . 0727 | . 0609 | . 0541 | . 0533 | . 0297 | . 0150 |
|  | 1.6 | . 3298 | . 1492 | . 0282 | . 0473 | . 0584 | . 0726 | . 0607 | . 0539 | . 0531 | . 0295 | . 0149 |
|  | 1.8 | . 3294 | . 1478 | . 0281 | . 0472 | . 0583 | . 0725 | . 0605 | . 0536 | . 0538 | . 0293 | . 0148 |
|  | 2.0 | . 3291 | . 1466 | . 0281 | . 0471 | . 0581 | . 0723 | . 0604 | . 0535 | . 0536 | . 0291 | . 0147 |
| 0.4 | 1.0 | . 3278 | . 1455 | . 0280 | . 0469 | . 0578 | . 0718 | . 0597 | . 0527 | . 0418 | . 0287 | . 0146 |
|  | 1.2 | . 3268 | . 1426 | . 0279 | . 0467 | . 0575 | . 0714 | . 0592 | . 0521 | . 0412 | . 0282 | . 0143 |
|  | 1.4 | . 3258 | . 1400 | . 0278 | . 0465 | . 0572 | . 0710 | . 0587 | . 0515 | . 0406 | . 0277 | . 0140 |
|  | 1.6 | . 3249 | . 1376 | . 0277 | . 0463 | . 0570 | . 0707 | . 0583 | . 0509 | . 0400 | . 0272 | . 0137 |
|  | 1.8 | . 3241 | . 1354 | . 0276 | . 0461 | . 0567 | . 0703 | . 0579 | . 0505 | . 0395 | . 0267 | . 0135 |
|  | 2.0 | . 3233 | . 1334 | . 0275 | . 0459 | . 0565 | . 0700 | . 0574 | . 0500 | . 0390 | . 0262 | . 0133 |
| 0.5 | 1.0 | . 3225 | . 1357 | . 0275 | . 0458 | . 0563 | . 0697 | . 0571 | . 0496 | .03c1 | . 0277 | . 0135 |
|  | 1.2 | . 3205 | . 1316 | . 0273 | . 0454 | . 0557 | . 0689 | . 0561 | . 0485 | . 0381 | . 0260 | . 0131 |
|  | 1.4 | . 3187 | . 1279 | . 0271 | . 0450 | . 0551 | . 0682 | . 0552 | . 0475 | . 0372 | . 0253 | . 0128 |
|  | 1.6 | . 3170 | . 1245 | . 0269 | . 0447 | . 0546 | . 0675 | . 0543 | . 0466 | . 0363 | . 0246 | . 0125 |
|  | 1.8 | . 3153 | . 1214 | . 0267 | .0444 | . 0539 | . 0668 | . 0530 | . 0457 | . 0354 | . 0240 | . 0122 |
|  | 2.0 | . 3138 | . 1186 | . 0266 | . 0439 | . 0536 | . 0662 | . 0527 | . 0459 | . 0345 | . 0234 | . 0119 |

TABLE III
BEAM CONSTANTS FOR BEAM WITH ONE PARABOLIC HAUNGH
(END B)


| Right Haunch |  | AngularFlexi-bilitiesCoef.x $\frac{L}{E I_{0}}$ | AngularCarry-OverValuesCoef. | Influence Values of Angular Load Functions; Coef. $\times \frac{\mathrm{PL}^{2}}{\mathrm{EI}_{0}}$ |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | $n$ n |  |
|  |  | 0.1 |  | 0.2 | 0.3 | 0.4 | 0.5 | 0.6 | 0.7 | 0.8 | 0.9 |
| $\beta$ | $\omega$ |  | $\mathrm{F}_{\mathrm{BA}}$ | $\mathrm{G}_{\mathrm{BA}}$ | $\tau_{B A}$ | $\tau_{\text {BA }}$ | $\tau_{\text {BA }}$ | $\tau_{B A}$ | $\tau_{\mathrm{BA}}$ | $\tau_{\mathrm{BA}}$ | $\tau_{B A}$ | $\tau_{B A}$ | $\tau_{\text {BA }}$ |
| 0.3 | 1.0 |  | . 2186 | . 1540 | . 0152 | . 0295 | . 0417 | . 0509 | . 0562 | . 0564 | . 0506 | . 0382 | . 0207 |
|  | 1.2 | . 2082 | . 1523 | . 0150 | . 0292 | . 0412 | . 0502 | . 0556 | . 0554 | . 0494 | . 0371 | . 0200 |
|  | 1.4 | . 1993 | . 1507 | . 0149 | . 0288 | . 0407 | . 0496 | . 0546 | . 0545 | . 0483 | . 0361 | . 0193 |
|  | 1.6 | . 1916 | . 1492 | . 0147 | . 0285 | . 0403 | . 0490 | . 0538 | . 0535 | . 0472 | . 0351 | . 0186 |
|  | 1.8 | . 1848 | . 1478 | . 0146 | . 0283 | . 0398 | . 0484 | . 0531 | . 0527 | . 0463 | . 0341 | . 0180 |
|  | 2.0 | . 1788 | . 1466 | . 0145 | . 0280 | . 0395 | . 0480 | . 0525 | . 0520 | . 0454 | . 0331 | . 0174 |
| 0.4 |  | . 1913 |  |  |  |  |  |  |  |  | . 0336 |  |
|  | 1.2 | . 1780 | . 1426 | . 0141 | . 0272 | . 0383 | . 0463 | . 0573 | . 0496 | . 0431 | . 0319 | . 0170 |
|  | 1.4 | . 1684 | . 1400 | . 0138 | . 0267 | . 0375 | . 0453 | . 0492 | .0480 | . 0414 | . 0304 | . 0161 |
|  | 1.6 | . 1594 | . 1376 | . 0136 | . 0262 | . 0368 | . 0443 | . 0480 | . 0466 | . 0399 | . 0291 | . 0154 |
|  | 1.8 | . 1515 | . 1354 | . 0133 | . 0257 | . 0361 | . 0435 | . 0469 | . 0452 | . 0385 | . 0279 | . 0148 |
|  | 2.0 | . 1446 | . 1334 | . 0131 | . 0254 | . 0355 | . 0427 | . 0459 | . 0440 | . 0372 | . 0267 | . 0143 |
| 0.5 | 1.0 | . 1686 | . 1357 | . 0134 | . 0258 | . 0362 | . 0436 | . 0469 | . 0456 | . 0397 | . 0294 | . 0160 |
|  | 1.2 | . 1549 | . 1316 | . 0130 | . 0250 | . 0350 | . 0419 | . 0450 | . 0432 | . 0373 | . 0276 | . 0148 |
|  | 1.4 | . 1433 | . 1279 | . 0126 | . 0243 | . 0339 | . 0406 | . 0432 | . 0411 | . 0352 | . 0258 | . 0137 |
|  | 1.6 | . 1335 | . 1245 | . 0123 | . 0235 | . 0329 | . 0391 | . 0415 | . 0392 | . 0333 | . 0242 | . 0128 |
|  | 1.8 | . 1249 | . 1214 | . 0119 | . 0230 | . 0319 | . 0379 | . 0399 | . 0375 | . 0315 | . 0228 | . 0120 |
|  | 2.0 | . 1165 | . 1186 | . 0117 | . 0224 | . 0311 | . 0364 | . 0385 | . 0358 | . 0300 | . 0217 | . 0115 |

## CHAPTER V

## BEAM WITH TWO SYMMETRICAL PARABOLIC HAUNCHES

The angular constants for a prismatic member with two symmetrical parabolic haunches, simply supported at both ends,are derived in a general form (Fig. 5-1).


Fig. 5-1
a) Angular Flexibilities and Carry Over Values. If the beam shown in Fig. (5-1) is acted upon by a unit moment applied at $A$, the end slopes of the elastic curve at $A$ and $B$ are called the angular flexibility $F_{A B}$ and the angular carry over value $G_{B A}$, respectively. Fig. 5-2 shows the bending moment, the elastic curve, the string polygon, and the conjugate beam. The elastic weights $\phi_{A}, \phi_{C}, \varnothing_{D}, \phi_{B}$ shown in Fig. (5-2) and defined by Eq. (1-1) are:

$$
\begin{align*}
& \phi_{A}=F_{A C}+(1-\beta) G_{C A} \\
& \phi_{C}=G_{A C}+(1-\beta)\left(F_{C A}+F_{C E}\right)+\beta G_{E C}  \tag{5-1}\\
& \phi_{E}=(1-\beta) G_{C E}+\beta\left(F_{E C}+F_{E B}\right) \\
& \phi_{B}=\beta G_{E B}
\end{align*}
$$



Bending Moment Diagram


Elastic Curve


String Polygon


F1g. 5-2

The angular flexibility $\mathrm{F}_{\mathrm{AB}}$ is the left reaction of the conjugate beam (Fig. 5-2).

$$
\begin{aligned}
F_{A B} & =\varnothing_{A}+(1-\beta) \emptyset_{C}+\beta \phi_{E} \\
& =F_{A C}+2(1-\beta) G_{A C}+\left(1-2 \beta+2 \beta^{2}\right) F_{C A}+\frac{L \alpha}{3 E I_{0}}\left(1-\beta+\beta^{2}\right) \quad(5-2 a)
\end{aligned}
$$

or in terms of $Q s$

$$
\begin{align*}
F_{A B}=\frac{L \beta}{E I_{0}}\left[\left(1-2 \beta+2 \beta^{2}\right) Q_{0}^{(L \cdot \beta)}+2 \beta(1-2 \beta)\right. & \left.Q_{1}^{(L \beta)}+2 \beta^{2} Q_{2}^{(L \beta)}\right] \\
& +\frac{L \alpha}{3 E I_{0}}\left(1-\beta+\beta^{2}\right) . \tag{5-2b}
\end{align*}
$$

The carry over value $G_{B A}$ is the right reaction of the conjugate beam (Fig. 5-2).

$$
\begin{align*}
G_{B A} & =\beta \phi_{C}+(1-\beta) \phi_{E}+\phi_{B} \\
& =2 \beta(1-\beta) F_{C A}+2 \beta G_{A C}+\frac{I \alpha}{6 E I_{0}}\left(1+2 \beta-2 \beta^{2}\right) \tag{5-3a}
\end{align*}
$$

or in terms of $Q \mathrm{~s}$

$$
\begin{align*}
& G_{B A}=\frac{L L \beta^{2}}{E I_{0}}\left[(1-\beta) Q_{0}^{(L \beta)}-(1-2 \beta) Q\right. \\
& 1(L \beta)  \tag{5-3b}\\
& 1\left.\beta Q_{2}^{(L \beta)}\right] \\
&+\frac{L \alpha}{6 E I_{0}}\left(1+2 \beta-2 \beta^{2}\right)
\end{align*}
$$

If the beam shown in Fig. (5-1) is acted upon by a unit moment applied at $B$, the end slopes of the elastic curve at $A$ and $B$ are
called the angular carry over value $G_{A B}$ and the angular flexibility $F_{B A}$, respectively. Because the beam shown in Fig. (5-1) is symmetrical,

$$
\begin{equation*}
F_{B A}=F_{A B} \tag{5-4}
\end{equation*}
$$

and

$$
\begin{equation*}
G_{A B}=G_{B A} . \tag{5-5}
\end{equation*}
$$

b) Angular Load Functions If the beam shown in Fig. (5-1) is acted upon by a unit load moving gradually from $A$ to $B$, the end slopes of the elastic curve at $A$ and $B$ are called the influence values of angular load function. The procedure for determining these influence values is similar to that for a beam with one haunch. The general formulas of the influence values will depend upon the position of the load.

The beam shown in Fig. (4-3) is acted upon by a unit load applied in the intervening straight part of the member. The formula will be derived by adding together two partial effects, (a) and (b). The angular load functions due to the bending moment of part (a) follow:

$$
\left.\begin{array}{l}
\tau_{A B}^{\prime}=\operatorname{Lr}_{2} F_{A B}  \tag{5-6}\\
\tau_{B A}^{\prime}=\operatorname{Ln} G_{B A}
\end{array}\right\}
$$

The elastic weights $\varnothing_{A}, \phi_{C}, \varnothing_{p}$ due to the bending moment of part (b) are:

$$
\left.\begin{array}{l}
\phi_{A}=-\operatorname{Ln} F_{A C}-L \alpha k G_{C A}  \tag{5-7}\\
\phi_{C}=-\operatorname{Ln} G_{A C}-L \alpha k\left(F_{C A}+F_{C P}\right) \\
\phi_{P}=-\operatorname{L\alpha k} G_{C P}
\end{array}\right\}
$$



The angular load functions $\tau_{A B}^{\prime \prime}, \tau_{B A}^{\prime \prime}$ are the reactions of the conjugate beam at left and right ends, respectively.

$$
\begin{align*}
\tau_{A C}^{\prime \prime} & =\phi_{A}+(1-\beta) \phi_{C}+n^{\prime} \phi_{P} \\
& =-L n F_{A C}-L(2 n-\beta-n \beta) \quad G_{A C}-L \alpha_{k}(1-\beta) F_{C A} \\
& -\frac{(L \alpha k)^{2}}{6 E I}(3-2 \beta-n)  \tag{5-8}\\
\tau_{B A}^{\prime \prime} & =\beta \phi_{C}+n \phi_{P} \\
& =-L \beta n G_{A C}-L \beta \alpha k F_{C A}-\frac{(L \alpha k)^{2}}{6 E I}(2 \beta+n) .
\end{align*}
$$

The angular load functions $\tau_{A B}, \tau_{B A}$ may be obtained by adding the Eqs. (5-6) and (5-8) :

$$
\begin{aligned}
& \begin{aligned}
\tau_{A B}=\operatorname{Ln} F_{A B}-\left\{\begin{array}{l}
L \beta \\
E I_{0}
\end{array} \alpha k(1-\beta): Q_{0}^{(L \beta)}\right. & \left.+\left(n^{\prime}+2 \alpha k\right) \beta Q_{1}^{(L \beta)}+\beta^{2} \cdot Q_{2}^{(L \beta)}\right] \\
& \left.+\frac{(L \alpha k)^{2}}{6 E I_{0}}(3-2 \beta-n)\right\}
\end{aligned} \\
& \tau_{B A}=\operatorname{Ln} G_{B A}-\left\{\frac{(L \beta)^{2}}{E I_{0}}\left[\alpha k Q_{0}^{(L \beta)}+(\beta-\alpha k) Q_{1}^{(L \beta)}-\beta Q_{2}^{(L \beta)}\right]\right. \\
& \left.+\frac{(L \alpha k)^{2}}{6 E I_{0}}(2 \beta+n)\right\}_{(5-9)} \text {. }
\end{aligned}
$$

If the beam shown in Fig. (5-1) is acted upon by a unit load applied at a point within one of the haunches (Fig. 5-4), the angular load functions due to the bending moment of part (a) follow:

$$
\left.\begin{array}{l}
\tau_{\mathrm{AB}}^{\prime}=\operatorname{Ln} \mathrm{F}_{\mathrm{AB}}  \tag{5-10}\\
\tau_{\mathrm{BA}}^{\prime}=\operatorname{Ln} G_{\mathrm{BA}}
\end{array}\right\}
$$

The elastic weights $\phi_{A}, \varnothing_{C}, \emptyset_{E}, \varnothing_{D}$ due to the bending moment of part (b) are:

$$
\begin{align*}
& \phi_{A}=-\operatorname{Ln} F_{A C}-L(n-\beta) G_{C A} \\
& \phi_{C}=-\operatorname{Ln} G_{A C}-L(n-\beta)\left(F_{C A}+F_{C E}\right)+L \beta k G_{E C}  \tag{5-11}\\
& \phi_{E}=-\operatorname{Ln}(n-\beta) G_{C E}-L \beta k\left(F_{E C}+F_{E P}\right) \\
& \phi_{P}=-L \beta k G_{E P} .
\end{align*}
$$

The angular load functions $\tau_{A B}^{\prime \prime}, \tau_{B A}^{\prime \prime}$ are the reactions of the conjugate beam at left and right ends, respectively:

$$
\begin{aligned}
\tau_{A B}^{\prime \prime}= & \phi_{A}+\beta \phi_{D}+n^{\prime} \phi_{C} \\
= & -\left[L n F_{A C}+L(2 n-\beta n-\beta) G_{A C}+L(n-\beta)(1-\beta) F_{C A}\right. \\
& \left.+\frac{L_{D}^{2}}{E I_{O}} \frac{\alpha}{3}\left(3 n-1-2 \beta+2 \beta^{2}\right)+L \beta^{2} k F_{E P}+L n^{\prime} k G_{E P}\right] \\
\tau_{B A}^{\prime \prime}= & \beta \phi_{C}+(1-\beta) \phi_{E}+n \phi_{P} \\
= & -\left\{L \beta n G_{A C}+L \beta(n-\beta) F_{C A}+\frac{L^{2}}{E I_{0}} \frac{\alpha}{3}\left[3 n-2\left(1-\beta+\beta^{2}\right)\right]\right. \\
& \left.+L(1-\beta) \beta k F_{E P}+\operatorname{Ln} B k G_{E P}\right\}
\end{aligned}
$$

The angular load functions $\tau_{A B}, \tau_{B A}$ may be obtained by adding the
Eq. (5-10) and (5-12) together.

$$
\begin{aligned}
& \tau_{A B}=\operatorname{Ln} F_{A B}-\left\{\frac{L^{2} \beta}{E I_{0}}\left[(n-\beta)(1-\beta) Q_{0}^{(L \beta)}+\beta(n+2 \alpha) Q_{1}^{(L \beta)}+\beta^{2} Q(L \beta)\right]\right. \\
& +\frac{L^{2}}{E I_{0}} \frac{\alpha}{3}\left(3 n-1-2 \beta+2 \beta^{2}\right)+\frac{(L \beta k)^{2}}{E I_{0}}\left[\begin{array}{c}
\beta k Q \\
0
\end{array}(L \beta k)+\left(n^{\prime}-2 \beta k\right) Q(L \beta k)\right. \\
& +\left(\beta k-n^{\prime}\right) Q_{2}^{\left(\frac{1}{2} \beta k\right)} \\
& \text { (5-13) }
\end{aligned}
$$



$$
\begin{aligned}
& \tau_{B A}=\operatorname{Ln} G_{B A}-\left\{\frac{(L, \beta)^{2}}{E I_{0}}\left[(n-\beta) \underset{0}{Q}(L \beta)+(2 \beta-n) Q_{1}^{(L \beta}\right)-\beta Q_{2}^{(L \beta)}\right. \\
& +\frac{L^{2}}{E I_{0}} \frac{\alpha}{3}\left[3 n-2\left(1-\beta+\beta^{2}\right)\right] \\
& \left.+\frac{(L \beta k)^{2}}{E I_{0}}\left[\begin{array}{c}
(1-\beta) \\
Q \\
0
\end{array} \frac{(L \beta k)}{0}+(2 \beta+n-2) \underset{2}{(L \beta k)}-\left(n^{\prime}-\beta\right) Q \underset{2}{(L \beta k)}\right]\right\} . \\
& \text { (5-13) }
\end{aligned}
$$

The values of beam constants are computed and listed in Table IV.

TABLE IV
BEAM CONSTANTS FOR BEAM WITH
TWO SYMIETRICAL PARABOLIC haUNCHES (END A)


| Haunch |  | AngularFlexi-bilitiesCoef.x $\frac{L}{E I_{0}}$ | Angular <br> Carry- <br> Over <br> Factors <br> Coef.x $\frac{E I_{0}}{}$ | Influence Values of Angular Load Function, Coef. $\times \frac{\mathrm{PL}^{2}}{\mathrm{EI}_{0}}$ |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
|  |  | n |  |
|  |  | 0.1 |  | 0.2 | 0.3 | 0.4 | 0.5 | 0.6 | 0.7 | 0.8 | 0.9 |
| $\beta$ | $\omega$ |  | $\mathrm{F}_{\mathrm{AB}}$ | $\mathrm{G}_{\mathrm{AB}}$ | $\tau_{\text {AB }}$ | ${ }^{\tau_{A B}}$ | $\tau_{\text {AB }}$ | $\tau_{A B}$ | $\tau_{A B}$ | $\tau_{A B}$ | $\tau_{A B}$ | $\tau_{A B}$ | $\tau_{A B}$ |
| 0.2 | 1.0 |  | . 2503 | . 1547 | . 0238 | . 0431 | . 0552 | . 0601 | . 0592 | . 0539 | . 0437 | . 0302 | . 0154 |
|  | 1.2 | . 2424 | . 1530 | . 0232 | . 0424 | . 0545 | . 0595 | . 0586 | . 0527 | . 0428 | . 0300 | . 0153 |
|  | 1.4 | . 2357 | . 1514 | . 0222 | . 0416 | . 0539 | . 0589 | . 0582 | . 0524 | . 0426 | .0297 | . 0151 |
|  | 1.6 | . 22297 | . 1500 | . 0221 | . 0410 | . 0535 | . 0585 | . 0579 | . 0521 | . 0422 | . 0295 | . 0149 |
|  | 1.8 | . 2245 | . 1487 | . 0217 | . 0408 | . 0529 | . 0581 | . 0575 | . 0518 | . 0420 | . 0292 | . 0148 |
|  | 2.0 |  | . 1475 | . 0214 | . 0401 | . 0525 | . 0576 | . 0571 | . 0515 | . 0418 |  | . 0147 |
| 0.3 | 1.0 | . 2163 |  | . 0204 |  | . 0499 | . 0553 | . 0551 | . 0496 | . 0401 | . 0278 | . 0143 |
|  | 1.2 | . 2054 | . 1378 | . 0196 | . 0365 | . 0485 | . 0543 | . 0540 | . 0486 | . 0392 | . 0271 | . 0138 |
|  | 1.4 | . 1961 | . 1346 | . 0199 | . 0353 | . 0473 | . 0532 | . 0530 | . 0477 | . 0385 | . 0266 | . 0134 |
|  | 1.6 | . 1880 | . 1313 | . 0182 | . 0343 | . 0462 | . 0521 | . 0520 | . 0469 | . 0378 | . 0260 | . 0131 |
|  | 1.8 | . 1809 | . 1289 | . 0176 | . 0332 | . 0451 | . 0511 | . 0512 | . 0461 | . 0371 | .0254 | . 0128 |
|  | 2.0 | . 1746 | . 1265 | . 0171 | . 0323 | . 0442 | . 0502 | . 0504 | . 0455 | . 0366 | . 0250 | . 0126 |

## CHAPTER VI

CONCLUS IONS

The general formulas for the angular flexibilities, carry over values and load functions for beams with parabolic haunches by means of string polygon are derived in this thesis.

The influence of parabolic haunches is expressed by means of Ritter's approximation. The final formulas are expressed in terms of the most common values of parameters $\beta$ and $\omega$, and the results are recorded in tables.

These tables are compared with results published elsewhere (3). The maximum error which occur obtained by using this approximation are 8 percent in the case of beams with one haunch and 7 percent in the case of beams with two symmetrical haunches.

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