# NONLINEAR BEHAVIOR OF REINFORCED CONCRETE BEAMS BY THE FINITE ELEMENT METHOD

By

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Thesis Approved:

Thesis Adviser

burnt. ,

Dean of the Graduate College

To my wife,

Elizabeth

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# NOMENCLATURE

А	area of concrete element
[a]	rectangular matrix: function of position coordinates of
	a point within an element
[b]	strain displacement transformation matrix
	square matrix of elastic constants
E	modulus of elasticity
$\{e\}$	column vector of total strains within an element
G	shear modulus
H'	strain hardening parameter (slope of effective stress-
	effective plastic strain function)
<sup>1</sup> 2	second stress invariant
$J_2$	second strain invariant
R	coordinate transformation matrix for concrete elements
S	surface on the boundary of a continuum
Т	coordinate transformation matrix for steel and bond
	elements
$\{U\}$	column vector of nodal displacements in the global coor-
	dinate system
$\left\{ U \right\}$	column vector of nodal displacements in the local system
u	displacement function
V	volume of concrete elements
$x \\ y$	coordinates of nodes

δ	variational operator
Δ	symbol for incremental values
$\left\{ \Delta \epsilon_{o}^{}, p \right\}$	initial strain increments
$\left\{ \Delta \epsilon_{\mathbf{p}} \right\}$	plastic strain increments
$\{\epsilon\}$	column vector elastic strains within an element
ε <sub>o</sub>	uniaxial strains
$\epsilon_{ m eff}$	effective strains
ν	Poisson's ratio
$\{\sigma\}$	column vector of stresses within an element
σ	uniaxial stress
$\sigma_{eff}$	effective stress
$ au_{ m oct}$	octahedral shear stress

#### CHAPTER I

#### INTRODUCTION

#### 1.1 <u>General Discussion</u>

The behavior of reinforced concrete structures subjected to various types of loads has been studied extensively during the past few decades. In spite of many efforts, no basic analytical approach has been developed to determine accurately the stress distribution in the concrete and the steel. This is mainly due to the fact that the constitutive relations for concrete depend on a number of factors such as the size and shape of the structure, the size, the material properties and the composition of the aggregate, and the rate and duration of loading. Furthermore, the tensile strength of concrete is much lower than its compressive strength. Therefore, additional difficulties arise from the continuing change in structural configuration caused by cracks in the concrete.

It is even more difficult to express the many different geometric shapes of the stress-strain curves for steel in analytical form. Here the manufacturing process and the choice of alloys have the most significant influence on the material properties. Finally, time-dependent effects on concrete strains, steel relaxation and complicated laws of interaction between concrete and reinforcements render a closed-form solution practically impossible. It is, therefore, necessary to utilize empirical laws obtained from extensive test data.

Present methods of analysis or design are based on assumptions which allow the application of the fundamental principles of continuum mechanics and empirical or simplified constitutive equations. Two different approaches are commonly used in the design of reinforced concrete structures. Both methods assume a perfect bond between steel and concrete and neglect the tensile resistance of concrete. The first assumption allows the use of the classical Navier-Bernoulli stipulation for planes perpendicular to the member axis. According to this assumption, these planes remain plane and perpendicular to the centroidal axis during the entire load history. Experiments on reinforced concrete beams have confirmed that the assumed strain distribution actually deviates very little from the real strain condition, provided that good bonding exists.

The main difference in the two methods lies in the choice of stress-strain relations. The "Working Stress Method" (1) utilizes linear material laws. Since concrete behaves elastically only as long as the maximum compressive stress is less than about half the ultimate strength, this approach has failed to give correct pictures of the stress distribution at high loads.

The "Ultimate Strength Method" (1), on the other hand, is based upon stress conditions just before failure occurs. It may appear essential to use realistic constitutive relations at these high stress levels. However, this is not the case because the geometric shape of the stress distribution has little effect on the location and magnitude of the resultant compressive force in the concrete. The real stress situation is therefore usually approximated by an equivalent rectangular or trapezoidal stress block.

A lthough both methods are of chief importance in design, neither one is of much help in studies of the nonlinear behavior of reinforced concrete beams. Such investigations are extremely involved due to previously stated reasons. Any reliable approach must therefore resort to numerical methods. With the introduction of the finite element technique to be discussed subsequently, such an analysis procedure has been established for the solution of complex problems of continuum mechanics. The application of this method results in a large system of linear, simultaneous equations which can be solved very efficiently on digital computers. Nonlinear problems introduce no new difficulties, since they can be treated either by iteration or as a sequence of consecutive linear problems.

# 1.2 Purpose and Scope of This Study

The purpose of this study is to develop a reliable tool for the analytical study of reinforced concrete members through their entire elastic, inelastic, and ultimate ranges.

The main emphasis is placed on the behavior in the inelastic range. Consequently, the problem approach is based upon nonlinear constitutive relations for steel, concrete, and bond. Nonlinearities introduced through the change in geometry are not included since the beams are assumed to have failed long before large displacements develop. Also, time dependent effects on concrete strains (such as creep and relaxation of reinforcements) are neglected. The loading history is restricted to monotonically increasing static loads.

After each load increment, the stress and strain distributions will be calculated. The arrangement of steel components is kept flexible in order to allow the study of various types of reinforcements.

# 1.3 Historical and Literature Review

The successful application of matrix analysis methods to materially nonlinear framed structures by Wilson (2) in 1960, and Goldberg and Richard (3) in 1963 demonstrated the feasibility of the finite element method for the solution of nonlinear problems. Wilson subsequently extended the incremental load procedure to a class of two-dimensional, nonlinear structures (4) in 1963. In the same report an iterative technique similar to the Newton-Raphson Method was applied to in-plane loaded thin plates with bilinear constitutive relations.

Argyris (5) and Denke (6), in 1964, adapted the matrix force method to elasto-plastic problems. Comprehensive presentations of the elasto-plastic displacement method were given by Pope (7) in 1965 for plane stress and plane strain states and by Argyris (8, 9) for threedimensional states of stress. Both publications distinguish clearly between the two basic incremental procedures referred to as "Initial Strain Method" and "Tangent Modulus Method."

The "Initial Strain Method" was developed in matrix form by Argyris (8, 9). It involves approximating the change in plastic strain during each load increment. These plastic strains are then used as initial strains to reevaluate the stress distribution. Therefore, this procedure requires iterations in each loading step. The "Tangent Modulus Technique" makes use of incremental stiffness matrices which are derived from well-known incremental stress-strain relations. For strain-hardening material, the stiffness matrices must be modified after each load increase. A partial stiffness method for elasto-plastic problems based on the "Tangent Modulus Approach" was first proposed by Marcal (10) in 1965 and later modified for use in the finite element method by Marcal and King (11). These papers state the necessary equations in matrix form and suggest the sequence of steps suitable for digital computation.

In his classical treatise, Zienkiewicz (12) presents an excellent summary of these fundamental matrix methods and also presents Wilson's "Direct Iterative Approach" as a third basic technique. The amount of research conducted in the area of nonlinear analysis by finite elements has increased rapidly since these initial efforts. Therefore, only the most significant publications pertinent to this study will be mentioned. In general, recent investigations have only refined the earlier formulations of the elasto-plastic problem.

Felippa's paper (13) can be considered as one of the early attempts to introduce refinements into the matrix methods for linear and nonlinear analysis of two-dimensional structures. Other planar problems were solved by Akyuz (14) and Akyuz and Merwin (15). Special attention was given in these publications to the computational difficulties arising from the repeated solution of simultaneous equations. A half-step method related to the Runge-Kutta procedure was applied to improve the accuracy. The comparative study by Marcal (16) in 1968 revealed that the Initial Strain Method fails for the case of elastic-perfectly-plastic material. Otherwise, the two incremental techniques were found to provide very similar results. Another contribution to the topic was presented by Marcal (17) in 1969. At the same time, Yamada (18) gave a general review of Japanese developments in the field of elasto-plastic matrix analysis. His paper contains an incremental stress-strain matrix for anisotropic materials and shows several practical applications of the step-by-step approach.

A variation of the Initial Strain Method based on known stress functions was proposed by Yamada et al. (19) in 1968. However, because the publication of their paper was delayed until 1969, it appears that Zienkiewicz, Valliappan and King (20) should earn full credit for the development of the so-called "Initial Stress Method." This new technique makes use of the fact that plastic strain increments prescribe uniquely the stress system, even in the case of an ideally plastic material. With this in mind, Zienkiewicz et al. were able to retain the advantages of the Initial Strain Method for which the matrix of elastic constants remains unchanged during the loading history. Probably the most comprehensive survey concerning nonlinear structural analysis techniques was made by Oden (21) in 1969. The main solution methods for both geometrically and materially nonlinear structures are discussed and presented in tensor form. Furthermore, the incremental stiffness approach first suggested by Pope (7) is generalized. The paper also includes an extensive list of selected references.

Despite the fact that finite element methods are highly suited for stress analyses, relatively few studies have adopted these techniques to investigate the behavior of concrete structures. Rashid (22) reported in 1966 the results of a two-dimensional finite element scheme used to analyze a prestressed concrete pressure vessel. In order to obtain a realistic model of this composite, heterogeneous, axisymmetric structure, three kinematically dissimilar elements were introduced to simulate the concrete, reinforcements and the steel liner. The program was later modified by Rashid (23) to include cracks in the concrete and the effects of plastic deformation in the steel components. As a special feature, Rashid proposed to treat the influence of a crack as a mechanism that changes the behavior of continuous elements from isotropic to orthotropic.

An alternative approach, suggested in 1967 by Scordelis and Ngo (24), introduced complete crack patterns by separating interelement boundaries. This study also included the simulation of bonds between reinforcement and concrete. Finally, the disadvantage of two-dimensional approximations has been overcome through the implementation of the "SAFE-3D" computer program, developed by Cornell et al. (25). This program was used by Corum and Krishnamurthy (26) to investigate a series of models of prestressed reactor vessels. It uses tetrahedral concrete elements, uniaxial bars, and triangular membrane steel components. As expected, the three-dimensional model provided much better results. However, a significant increase in computer time resulted from use of three-dimensional elements. Quite a different approach was taken by Cervenka (27) in his study entitled "Inelastic Finite Element Analysis of Reinforced Concrete Panels under In-Plane Loading." No individual reinforcement bars were considered. Instead, the total steel area was distributed over the quadrilateral element. The cracked state then could be visualized as a planar lattice structure.

# 1.4 Problem Approach

The solution method used herein is a combined iterative and stepby-step procedure based upon the matrix displacement method. The structure is analyzed as a plane stress problem. For each load increment, repeated elastic solutions are performed until the displacements meet a specified tolerance.

The mathematical model consists of an assemblage of triangular concrete plate elements, steel bar elements and bond links. The displacement fields are assumed to be linear for all three parts. The elastic constants (i. e., modulus of elasticity, Poisson's ratio, etc.) which are needed in the derivation of the elemental stiffness matrices are extrapolated from the pertinent uniaxial stress-strain curves. For all elements, these functions are approximated by piecewise linear polygons. The appropriate values of the material constants are found by entering the stress-strain diagram at the corresponding values of the principal strains.

A standard Gauss Elimination procedure is used to solve the equilibrium equations. Two computer programs were written to implement the method. Both provide stresses and strains in each element and the nodal displacements at all specified load levels.

A comparative study was made with the solutions presented by Scordelis for a simply supported beam (24). A second, more realistic problem was investigated to show the feasibility of the method to study crack propagation.

# CHAPTER II

# FINITE ELEMENT PROPERTIES

# 2.1 General

The finite element analysis of a continuum consists of three fundamental steps. First, the real structure is replaced by a suitable mathematical model. This is usually accomplished by dividing the original continuum into an assemblage of discrete elements. All elements are assumed to be interconnected at a discrete number of nodal points situated at the intersections of their fictitious boundaries. The second step is the formulation of the finite element characteristics.

In the matrix displacement approach, the material properties are described in the form of the elemental stiffness matrices. In recent years extensive research has been done in order to improve the various derivation procedures. Methods based upon energy theorems and related variational principles have been found to be the most satisfactory techniques. The foundation for such derivations is the assumption that an energy functional derived for the continuous system is equal to the same functional determined from the finite element model. The element properties can then be obtained by minimizing the functional through well-known variational methods.

Once the element properties of all the elements have been defined, the discrete system can be analyzed as a conventional structural

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. . problem. Hence, the last phase consists of a standard analysis of a structural system by means of suitable computer programs. In this chapter, the first two steps (i. e., the structural idealization and the evaluation of element characteristics) will be discussed. The mathematical models for concrete, steel reinforcements, and bond between the two materials are developed in section 2.4. The derivation of the solution procedure for linearly elastic problems and the necessary modifications for the nonlinear case will be presented in Chapter III.

# 2.2 Structural Idealization

In general, a reinforced concrete member must be considered as a three-dimensional, nonhomogeneous, nonisotropic, composite structure. The difficulties encountered in the solution of such structural systems have already been described in section 1.1. Clearly, a series of assumptions must be introduced in any solution procedure. The choice of suppositions is governed by the type of structure under consideration, the character of the results desired, and the numerical method utilized. In the case of reinforced concrete beams, simplifications concerning the type of structure are the most critical group. In order to obtain a reliable approximation, the model must include all physical constituents of the real composite structure. In addition, special attention should be given to the simulation of the interaction between the parts. A list of the necessary assumptions for the construction of a relevant model is set forth below.

The fo	ollow	ing	stipula	ations	may	be	rega	arded	as	prepa	ratory	re-
quirements	for a	ı po	ssible	applic	ation	of	the	finite	ele	ement	metho	d.

	<u>Real Structure</u>	Assumed Structure				
a.	Three-dimensional;	a. Two-dimensional (of the plane				
		stress type);				
b.	Nonhomogeneous components;	b. Homogeneous components;				
c.	Nonisotropic components;	c. Isotropic or orthotropic com-				
		ponents;				
d.	Random change in structural	d. Cracking predicted by princi-				
	configuration due to crack-	pal tensile stresses in the				
	ing;	concrete;				
e.	Continuous bond between con-	e. Discrete attachment between				
	crete and steel reinforce-	steel and concr <b>e</b> te via bond				
	ments; bond-slip;	links;				

f. Influence of time-dependent effects such as creep and relaxation.

f. Neglected.

The next set of assumptions concerns the selection of the finite elements and their individual properties.

Figure 1 shows a typical, singly-reinforced concrete beam under an arbitrary static, in-plane loading condition. The finite element idealization relevant to this study is displayed in Figure 2 in an exaggerated view.

Three kinematically and geometrically dissimilar elements have been chosen as basic components of the model. The entire concrete







body is divided into flat, triangular panels. Combined with the steel segments (represented by "two-force" members), they constitute the material part of the composite structure. The complicated phenomenon of bond interaction between concrete and steel is simulated by a dimensionless connecting device, called a linkage element. According to Scordelis (24), these bond links can be conceptually thought of as linear springs. Both steel reinforcements and the connecting elements have been extracted in Figure 4 for illustrative purposes. In the real assemblage, the nodes of the steel bars and the connecting springs originally occupy the same geometrical position as their corresponding concrete joints. Therefore, these nodes have the same global coordinates. However, topologically they must be treated as separate joints.

### 2.2.2 Concrete Elements

The concrete body can be subdivided in a number of ways. The most commonly used configurations are triangular, rectangular, and quadrilateral meshes. Rectangular elements provide slightly better results. However, triangular panels are preferred for problems with irregular boundaries. In early publications the stiffness matrices were derived by the so-called direct approach (28). Recently, descriptions of a number of refined elements have been published as a result of the implementation of variational techniques. An excellent summary may be found in Reference 29. In this study, the traditional, constantstrain, triangular panel (Figure 3a) has been adopted for two reasons. First, it is desirable to decrease the size of elements in the vicinity of large stress gradients. A gradual change in size can easily be accomplished in the case of triangular panels. The second criterion for











selecting constant-strain elements is reflected by the fact that yielding takes place throughout the whole element. Elements with nonuniform stress distributions are subject to local yielding which results in additional complications in determining the state of stress.

### 2.2.3 Steel Elements

The reinforcement occupies a relatively small volume compared to that of the concrete. It is therefore justifiable to idealize the steel tendons by simple two-force members (Figure 3b). The triangular model used by Scordelis was abandoned mainly because the very small vertical reinforcements would require a large number of additional elements or extremely slender triangles which are known to behave unsatisfactorily (30).

#### 2.2.4 Bond Links

To account for bond slip, the steel must be attached to the concrete by a special connection mechanism. The bond link (Figure 3c) is designed to allow for relative displacements between the steel bars and the concrete panels. As pointed out earlier, these elements are dimensionless because only their mechanical properties are of importance. Nevertheless, additional nodes must be provided to permit relative displacements between adjacent concrete and steel joints.

## 2.2.5 Displacement Functions

After the shape of an element has been chosen, all geometric relations can be established. The next logical step is to decide upon a suitable displacement function representing the deformation of the element. It should be noted that the degree of approximation which can be achieved depends very heavily on the element shape and the chosen deflection pattern. To ensure convergence, the assumed displacement function should resemble the real displacement distribution. According to Zienkiewicz (12), good deflection functions are obligated to satisfy the following five requirements:

- 1. Internal and interelement compatibility;
- 2. Linear dependence on nodal displacements;
- 3. Inclusion of rigid body displacements;
- 4. Uniform strain state;
- 5. Independence of the external frame of reference.

For all three elements utilized in this study, these criteria are satisfied by assumed linear displacement functions of the form

 $u_{x} = c_{1}x + c_{2}y + c_{3}$   $u_{y} = c_{4}x + c_{5}y + c_{6}$ for concrete panels (2.1)  $u_{1} = c_{7}x + c_{8}$ for steel and bond elements (2.2)

It can easily be shown that the assumed displacements vary linearly along the edges of the concrete panels and that they depend only on the displacement of the two vertices on that particular edge (31). This ensures displacement compatibility along the common boundary of two triangular elements.

The assumed linear deflection pattern for the "two-force" steel bars results not only in a compatible but also in an exact strain distribution, since the elements are one-dimensional.

On the basis of these chosen deformation functions, the kinematic relations (i.e., the strain-displacement equations) are derivable

through ordinary differentiation. To complete the preparations for the development of the stiffness matrices, the material laws for each element will be reviewed in the next section.

# 2.3 Constitutive Relations

# 2.3.1 General

The behavior of a material is characterized in the way it deforms under an imposed stress condition. It is therefore customary to express the material laws in the form of stress-strain curves. Two typical plots for a uniaxial, stress condition of nominal stress versus conventional strain for mild steel and concrete are shown in Figure 5. Both curves illustrate the complex, nonlinear character of the constitutive relations. Actually, the material characteristics can become even more complicated if the effects of time and temperature upon the rate of change of strains are included. It is therefore necessary to replace these empirical curves by mathematically defined expressions. The selection of an idealized stress-strain relationship depends upon several factors such as the nature of the problem, the kind of material, the type of load, desired accuracy, etc.

The most commonly used expression is the simple idealization known as Hooke's Law. In matrix form,

$$\left\{\sigma\right\} = \left[\mathbf{D}\right]\left\{\mathbf{e}\right\} \tag{2.3}$$

The  $\{e\}$  is the vector of total strains,  $\{\sigma\}$  designates the stress vector, and [D] is a square matrix containing the elastic constants. This linear relationship is, of course, very popular in engineering practice; however, its restriction to linearly elastic behavior must be remem-





bered. In an investigation concerning the nonlinear behavior, the entire stress-strain curve will be needed for the evaluation of the element characteristics. Analytical inelastic theories require that the constitutive relations may be replaced by reasonably simple continuous functions. The Ramberg-Osgood Law and the Bi-Linear Law are typical examples of such expressions (32). If more accurate idealizations are desired, the possibility of using a curve-fitting scheme always exists. In a numerical procedure, on the other hand, empirical data may be used directly in table-form. Values between discrete data points are easily calculated by means of suitable interpolation formulas. The approach adopted in this study is based upon linear interpolation; i. e., the stress-strain curves are replaced by a polygon (Figure 6).

After the idealized constitutive relations have been established, the elastic constants are available at any load level. Since the matrix  $\begin{bmatrix} D \end{bmatrix}$  contains "elastic constants" only, its derivation appears to be a straightforward procedure. This is true for the elastic interval. However, the evaluation of  $\begin{bmatrix} D \end{bmatrix}$  in the inelastic range presents some difficulties because of the biaxial state of stress in the concrete.

By methods well known from strength of materials, the biaxial state of stress can be reduced to two principal stresses acting at right angles to each other on an appropriately oriented elementary cube. Either, or both, of the principal stresses can be tension or compression.

In most cases only the uniaxial stress-strain relation of concrete is known from simple tests. To predict the elastic constants for a structure under a combined stress situation, it is necessary to relate the material properties to the uniaxial test parameters. Six different



;



Figure 6. Idealized Stress-Strain Curves

quantities (i.e., maximum principal stress, maximum shearing stress, maximum strain, total strain energy, strain energy of distortion, and octahedral shearing stress) are available to compare the multiaxial state of stress with a tensile specimen. When the specimen starts to yield (or fracture), all six quantities reach their limiting values simultaneously. In members under biaxial or triaxial states of stress, the limits usually do not occur at the same time. Since the type of failure of a concrete member is dependent upon many variables (i.e., state of stress, shape and size of structure, type and duration of loading, etc.), it is extremely difficult to choose the proper failure criterion. In spite of extensive and continuing research, no reliable theory for the selection of the proper failure mode has yet emerged. The highly nonhomogeneous nature of concrete and the phenomenon of microcracking are possible reasons for the insufficient reliability of these theories. The most commonly used criteria are the Maximum-Tension-Stress, the Mohr, and the Octahedral Shear Stress theories (33).

# 2.3.2 Proposed Idealization of Stress-Strain Characteristics

<u>Concrete</u>. In order to arrive at the proper  $\begin{bmatrix} D \end{bmatrix}$  matrix for concrete, this study used the following approach. Consider an element (shown in Figure 7) under an arbitrary strain condition  $\epsilon_u$  and  $\epsilon_v$ . From the stress-strain curve, Figure 8, it appears that two different Young's moduli,  $E_u$  and  $E_v$ , can be associated with the strains  $\epsilon_u$  and  $\epsilon_v$ , respectively. The material behavior is obviously different in these two directions; in other words, the structure may be thought of as anisotropic. It should be noted that this anisotropy is different from the term used in the theory of elasticity. There, an anisotropic body









is defined as a continuum with different values for E in at least two distinct directions. However, the elastic constants at each point in the structure are the same for one particular direction. Here, the behavior is assumed to change with the state of stress or strain at a point. If  $\epsilon_u$  and  $\epsilon_v$  are known, the corresponding values for the moduli of elasticity can be determined from the uniaxial stress-strain curve.

The constitutive relations for an anisotropic body are, in general, of the form

$$\{\sigma\} = [D_g] \{e\}$$
 (2.3a)

where  $\begin{bmatrix} D_g \end{bmatrix}$  is a symmetric matrix containing six independent, nonzero constants. For the principal axes of anisotropy, they reduce to four independent coefficients. The material is then referred to as "orthotropic" with respect to the axes u and v. Once again, the standard definition of orthotropy does not apply to the structures considered in this study.

For the principal axes of anisotropy, the constitutive equations become

$$\sigma_{u} = \frac{1}{1 - v_{uv}v_{vu}} (E_{u}\epsilon_{u} + v_{vu}E_{u}\epsilon_{v})$$

$$\sigma_{v} = \frac{1}{1 - v_{uv}v_{vu}} (v_{uv}E_{v}\epsilon_{u} + E_{v}\epsilon_{v}) \qquad (2.4)$$

$$\sigma_{uv} = G_{uv}\epsilon_{uv}$$

Under the assumption that the structure behaves locally as an orthotropic structure, one can assume that the principal axes of stress and strain coincide. Furthermore, these axes (1 and 2 in Figure 7) are taken as principal axes of anisotropy. With  $E_1$ ,  $v_{21}$  being the material constants in the direction of the first principal axis and  $E_2$ ,  $v_{12}$  being
the values for the second, Equation (2.4) now reduces to

$$\sigma_{1} = \frac{1}{1 - \nu_{12}\nu_{21}} (E_{1}\epsilon_{1} + \nu_{21}E_{1}\epsilon_{2})$$

$$\sigma_{2} = \frac{1}{1 - \nu_{12}\nu_{21}} (\nu_{21}E_{2}\epsilon_{1} + E_{2}\epsilon_{2}) \qquad (2.5)$$

$$\sigma_{12} = 0$$

Once the principal strains have been calculated, the corresponding maximum stresses may be evaluated directly as

$$\left\{\sigma\right\} = \left[D_{a}\right] \left\{e\right\}$$
(2.6)

where

$$\begin{bmatrix} D_{a} \end{bmatrix} = \frac{1}{1 - \nu_{12}\nu_{21}} \begin{bmatrix} E_{1} & \nu_{21}E_{1} & 0 \\ \nu_{12}E_{2} & E_{2} & 0 \\ 0 & 0 & 0 \end{bmatrix}$$
(2.7)

It must be kept in mind that such an approach is contingent upon the assumption that the principal axes exist. The conditions for the existence of principal directions are stated in Reference 34. Since the computer solution does not calculate the stresses in any other but the principal directions, no stress transformations are performed. Thus, the transformation of elastic constants for new coordinate systems can be omitted, and transformation of coordinate systems is performed on the entire element stiffness matrix.

Finally, it should be noted that Equation (2.5) is subject to an additional condition. The four material constants,  $E_1$ ,  $E_2$ ,  $v_{12}$  and  $v_{21}$ , are not independent. The additional relation may be obtained by comparison of the total work done on a differential element for two different

loading sequences. The resulting supplementary equation relates the Poisson's ratios to the moduli of elasticity as follows:

$$\frac{v_{12}}{v_{21}} = \frac{E_2}{E_1}$$
(2.8)

Substituting Equation (2.8) into Equation (2.5) yields the constitutive laws

$$\sigma_{1} = \frac{E_{1}}{1 - v_{12}v_{21}} (\epsilon_{1} + v_{21}\epsilon_{2})$$

$$\sigma_{2} = \frac{E_{1}}{1 - v_{12}v_{21}} (v_{21}\epsilon_{1} + n\epsilon_{2})$$
(2.9)
$$\sigma_{12} = 0$$

where

or

$$n = \frac{E_2}{E_1}$$
 (2.10)

In matrix form

$$\begin{bmatrix} \sigma_{1} \\ \sigma_{2} \\ 0 \end{bmatrix} = \frac{E_{1}}{1 - \nu_{12}\nu_{21}} \begin{bmatrix} 1 & \nu_{21} & 0 \\ \nu_{12} & n & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} \epsilon_{1} \\ \epsilon_{2} \\ 0 \end{bmatrix}$$
(2.11)
$$\{\sigma\} \qquad \qquad \begin{bmatrix} D_{a} \end{bmatrix} \quad \{\epsilon\}$$

<u>Steel and Bond Links</u>. Since in this study the steel elements will be considered as two-force members only, the uniaxial stress-strain curve may be used directly to determine the material properties. It should be mentioned that any other element would require more sophisticated tools for the evaluation of the material constants. A proper yield criterion and plastic stress-strain relations, such as the Prandtl-Reuss equations (32) would have to be adopted.

Here the incremental stress  $\Delta\sigma_{_{\rm X}}$  in the longitudinal direction is calculated by

$$\Delta \sigma_{\rm x} = E_{\rm s} \Delta \epsilon_{\rm x} \tag{2.12}$$

The appropriate modulus of elasticity is read from the uniaxial stressstrain curve for steel at the location of the total strain  $\epsilon_x$ . In the case of an elastic-perfectly-plastic material, the incremental stresses will become zero beyond the yield stress. However, the total stress  $\sigma_x$  is still available from the stress-strain curve.

A similar situation exists for the bond links. Again, the uniaxial stress-strain relations provide the material constant  $E_b$  directly. The same approach as used for steel members yields the stress increments  $\Delta \tau_b$  and the total bond stress  $\tau_b$  at any load level.

$$\Delta \tau_{\rm b} = E_{\rm b} \Delta_{\rm r} \tag{2.13}$$

 $\Delta_r$  denotes the relative displacement between a steel and the corresponding concrete node. Figure 9 displays some possible stress-relative displacement curves (from Reference (35)).

# 2.4 Development of Element Stiffness Matrices

2.4.1 General

A number of alternative methods are available for the calculation of element stiffness matrices. The variational approach based on the principle of minimum potential energy is adopted here. Since these methods are well established, a comprehensive repetition of the





procedure is omitted. References (31) and (36) contain excellent introductions to the variational treatment of the energy methods.

# 2.4.2 Matrix Formulation for the Plane Stress Case

The first step in determining the properties of the idealized element is to assume that the interior displacements  $\{u\}$  at any point are expressible in terms of the nodal displacements  $\{U\}$  by a set of equations given as

$$\left\{u\right\} = \left[a\right] \left\{U\right\}$$
(2.14)

 $\begin{bmatrix} a \end{bmatrix}$  is a rectangular matrix which is a function of the coordinates of the point under consideration. For discrete element systems, the matrix  $\begin{bmatrix} a \end{bmatrix}$  is an approximate expression. The total strain distribution  $\{e\}$  ( $\{e\}$  may include initial strains  $\{\epsilon_0\}$ ) within a particular element is obtained by differentiating Equation (2.14) which leads to the matrix equation

$$\left\{ \mathbf{e} \right\} = \left\{ \epsilon \right\} + \left\{ \epsilon_{\mathbf{o}} \right\} = \left[ \mathbf{b} \right] \left\{ \mathbf{U} \right\}$$
(2.15)

This expression replaces the kinematic relations used in the ordinary theory of elasticity.

Under the assumption that a unique matrix  $\begin{bmatrix} b \end{bmatrix}$  exists, the stresses may be determined from any conceivable constitutive relationship of the form

$$\left\{\sigma\right\} = \left[D\right] \left(\left\{e\right\} - \left\{\epsilon_{O}\right\}\right)$$
(2.16)

By substitution of Equation (2.15) into Equation (2.16)

$$\left\{\sigma\right\} = \left[D\right]\left[b\right]\left\{U\right\} - \left[D\right]\left\{\epsilon_{O}\right\}$$
(2.17)

 $\{\sigma\}$ , of course, represents the three stress components,  $\sigma_x$ ,  $\sigma_y$ ,  $\sigma_{xy}$  and [D] is a square matrix containing the elasticity constants, E, G,  $\nu$ , etc.

It is now possible to express the total energy functional in matrix form

$$\Pi = \sum_{n=1}^{N} \iiint_{V_{n}} \left(\frac{1}{2} \left\{e_{n}\right\}^{t} \left\{\sigma_{n}\right\} - \left\{f_{b,n}\right\}^{t} \left\{u_{n}\right\}\right) dV$$
$$- \sum_{n=1}^{N} \iint_{S_{\sigma_{1}n}} \left\{p_{n}\right\}^{t} \left\{u_{n}\right\} dS \qquad (2.18)$$

where

f<sub>b,n</sub> = prescribed body forces
 n = element index

 $p_n = prescribed surface tractions$ 

 $V_n =$  volume of element n

 $S_{\sigma, n}$  = portion of element surface over which the

surface tractions  $p_n$  are prescribed.

Substituting Equations (2.15) and (2.17) into Equation (2.18) yields the required expression for  $\Pi$ .

$$\Pi = \sum_{n=1}^{N} \iiint_{V_{n}} \left(\frac{1}{2} \left\{U_{n}\right\}^{t} \left[b_{n}\right]^{t} \left[D_{n}\right] \left[b_{n}\right] \left\{U_{n}\right\}$$
$$- \left\{U_{n}\right\}^{t} \left[b\right]^{t} \left[D_{n}\right] \left\{\epsilon_{o}\right\} - \left\{U_{n}\right\}^{t} \left[a\right]^{t} \left\{f_{n,o}\right\}\right) dV$$
$$- \sum_{n=1}^{N} \iint_{S_{\sigma_{1}}n} \left\{U_{n}\right\}^{t} \left[a\right]^{t} \left\{p_{n}\right\} dS \qquad (2.19)$$

Application of the principle of minimum potential energy

$$\delta \Pi = 0 \tag{2.20}$$

to Equation (2.19) will result in the desired stiffness matrix  $\begin{bmatrix} K \end{bmatrix}$ 

$$\begin{bmatrix} \mathbf{K} \end{bmatrix} = \iiint \begin{bmatrix} \mathbf{b}_n \end{bmatrix}^{\mathsf{t}} \begin{bmatrix} \mathbf{D}_n \end{bmatrix} \begin{bmatrix} \mathbf{b}_n \end{bmatrix} d\mathbf{V}$$
(2.21)

and three equivalent force vectors due to initial strain conditions, prescribed body forces and surface tractions (29).

Consideration will now be given to the three specific elements as shown in Figure 3. If the strain-displacement transformation matrix  $\begin{bmatrix} b_n \end{bmatrix}$  and the matrix of elastic constants  $\begin{bmatrix} D_n \end{bmatrix}$  are known, the stiffness matrix  $\begin{bmatrix} K_n \end{bmatrix}$  can be determined by evaluation of expression (2.21).

# 2.4.3 Triangular Concrete Panels

It is possible in this case to obtain the stiffness matrix  $\begin{bmatrix} K \\ n \end{bmatrix}$  directly in terms of global coordinates x and y. The assumed displacement function will be taken as

$$u_{x} = c_{1}x + c_{1}y + c_{3}$$

$$u_{y} = c_{4}x + c_{5}y + c_{6}$$
(2.22)

The six arbitrary constants,  $c_1, \ldots c_6$ , result from six boundary conditions involving the three vertices of the triangle, Figure 10. Upon substitution of the vertex coordinates into Equation (2.22), the displacement functions are obtained as follows:

$$u_{x}(x, y) = \frac{1}{2A_{n}} \left[ y_{32}(x - x_{2}) - x_{32}(y - y_{2}) \right] U_{1}$$
  
+  $\left[ -y_{31}(x - x_{3}) + x_{31}(y - y_{3}) \right] U_{3}$  (2.23a)  
+  $\left[ y_{21}(x - x_{1}) - x_{21}(y - y_{1}) \right] U_{5}$ 



Figure 10. Arbitrary Triangular Concrete Element

$$u_{y}(x, y) = \frac{1}{2A_{n}} \left[ y_{32}(x - y_{2}) - x_{32}(y - y_{2}) \right] U_{2}$$
  
+  $\left[ -y_{31}(x - x_{3}) + x_{31}(y - y_{3}) \right] U_{4}$  (2.23b)  
+  $\left[ y_{21}(-x_{1}) - x_{21}(y - y_{1}) \right] U_{6}$ 

where

$$A_{n} = x_{32}y_{21} - x_{21}y_{32}$$
 (2.24)

and

$$x_{ij} = x_i - x_j$$
  
 $y_{ij} = y_i - y_j$ 
(2.25)

Thus, matrix  $\begin{bmatrix} b_n \end{bmatrix}$  is a function of the vertex coordinates only and therefore is unique. The strain-displacement transformation matrix is obtained by differentiating Equation (2.23). Hence,

$$\begin{bmatrix} \epsilon_{x} \\ \epsilon_{y} \\ \epsilon_{xy} \end{bmatrix} = \frac{1}{2A_{n}} \begin{bmatrix} y_{32} & 0 & -y_{31} & 0 & y_{21} & 0 \\ 0 & -x_{32} & 0 & x_{31} & 0 & -x_{21} \\ -x_{32} & y_{32} & x_{31} & -y_{31} & -x_{21} & y_{21} \end{bmatrix} \begin{bmatrix} U_{1} \\ U_{2} \\ U_{3} \\ U_{4} \\ U_{5} \\ U_{6} \end{bmatrix}$$

$$(2.26)$$

or

$$\left\{ e \right\} = \left[ b_n \right] \left\{ U \right\}$$
 (2.27)

The assumption of linear displacement functions results, in this particular case, in a constant strain field. The compatibility equations

are therefore satisfied within each element. Furthermore, displacements along the interelement boundaries are linear functions of the corresponding vertices and are identical for adjacent edges.

Since the elements under consideration are of unit thickness, the expression (2. 21) reduces to

$$\begin{bmatrix} K_n \end{bmatrix} = \iint \begin{bmatrix} b_n \end{bmatrix}^t \begin{bmatrix} D_n \end{bmatrix} \begin{bmatrix} b_n \end{bmatrix} dxdy \qquad (2.28)$$

the integration being carried over the area of the triangle. Both matrices,  $\begin{bmatrix} b_n \end{bmatrix}$  and  $\begin{bmatrix} D_n \end{bmatrix}$ , are independent of the integration parameters and can be taken out of the integral sign. The integration then simply reduces to a matrix product of the form

$$\begin{bmatrix} K_n \end{bmatrix} = \begin{bmatrix} b_n \end{bmatrix} \begin{bmatrix} D_n \end{bmatrix} \begin{bmatrix} b \end{bmatrix} (1) A \qquad (2.29)$$

where A denotes the area of the triangle 1, 2, 3.

The resulting stiffness matrices for different matrices  $\begin{bmatrix} D_n \end{bmatrix}$  are tabulated in Appendix A.

# 2.4.4 Steel Bars

The derivation of  $\begin{bmatrix} K_n \end{bmatrix}$  for the linear steel elements is considerably less involved. Only one displacement function in the direction of the member axis is needed. It has been mentioned before that the assumption of linear functions of the form

$$\overline{u}_{x_1} = \overline{U}_1 + \frac{x_1}{L} (\overline{U}_3 - \overline{U}_1)$$
(2.30)

will provide the exact strain distribution.  $\overline{U}_1$  designates the displacements in local coordinates, Figure 11.

Upon differentiation of Equation (2.30), the longitudinal strain  $\epsilon_{x_1}$  becomes

$$\epsilon_{x_1} = \frac{\partial \overline{u}_{x_1}}{\partial x_1} = \frac{1}{L} (\overline{U}_3 - \overline{U}_1)$$
(2.31)

$$\begin{bmatrix} \epsilon_{x_1} \\ \epsilon_{y_1} \end{bmatrix} = \frac{1}{L} \begin{bmatrix} -1 & 0 & 1 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix} \begin{bmatrix} U_1 \\ U_2 \\ U_3 \\ U_4 \end{bmatrix}$$
(2.32)

and therefore,

$$\begin{bmatrix} b_{s} \end{bmatrix} = \frac{1}{L} \begin{bmatrix} -1 & 0 & 1 & 0 \\ 0 & -1 & 0 & 1 \end{bmatrix}$$
(2.33)

For one-dimensional elements,  $\begin{bmatrix} D_s \end{bmatrix}$  reduces to one term,  $E_s$ . The stiffness matrix then, after integrating over the length, can be written as

$$\begin{bmatrix} K_{s} \end{bmatrix} = \frac{AE_{s}}{L} \begin{bmatrix} 1 & 0 & -1 & 0 \\ 0 & 0 & 0 & 0 \\ -1 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix}$$
(2.34)

Young's modulus, naturally, must be chosen according to the prevailing strain condition in the member.

Although the stiffness matrix could be stated in the datum coordinate system directly, it is more convenient to develop the relations in local coordinates first and subsequently rotate the entire matrix into the global axes. According to Reference (2), the appropriate transformation is expressed by the matrix equation

$$\begin{bmatrix} \mathbf{K} \end{bmatrix}_{g} = \begin{bmatrix} \mathbf{T} \end{bmatrix}^{t} \begin{bmatrix} \mathbf{K} \end{bmatrix}_{\mathbf{L}} \begin{bmatrix} \mathbf{T} \end{bmatrix}$$
(2.35)

where  $\begin{bmatrix} T \end{bmatrix}$  is an orthogonal matrix which relates the nodal displacements  $\{U\}$  in the global system to the local deflections  $\{\overline{U}\}$  in the following manner.  $\theta$  denotes the angle between the local and global x-axis (Figure 11).

$$\left\{ \overline{\mathbf{U}} \right\} = \left[ \mathbf{T} \right] \left\{ \mathbf{U} \right\}$$
(2.36)  
$$\left[ \mathbf{T} \right] = \begin{bmatrix} \cos\theta & \sin\theta & | & 0 & 0 \\ -\sin\theta & \cos\theta & | & 0 & 0 \\ \hline 0 & 0 & | & \cos\theta & \sin\theta \\ \hline 0 & 0 & | & -\sin\theta & \cos\theta \end{bmatrix}$$
(2.37)

2.4.5 Bond Links

Finally, the derivation of the bond link stiffness matrix follows closely the procedure set forth in section 2.4.4. Consider a linkage element oriented at an arbitrary angle  $\theta$  relative to the global axes x and y, Figure 12. Let the springs in the  $x_1$  and  $y_1$  directions have stiffness coefficients  $k_1$  and  $k_2$ . Hence, the stress-strain relations in matrix notation simply become

$$\left\{ \sigma \right\} = \begin{bmatrix} D_{b} \end{bmatrix} \left\{ \epsilon \right\}$$

$$\left[ \begin{array}{c} \sigma_{x_{1}} \\ \sigma_{y_{1}} \end{array} \right] = \begin{bmatrix} k_{1} & 0 \\ 0 & k_{2} \end{bmatrix} \begin{bmatrix} \epsilon_{x_{1}} \\ \epsilon_{y_{1}} \end{bmatrix}$$

$$(2.39)$$

where  $\epsilon_{x_1}$  and  $\epsilon_{x_2}$  are the relative displacements between the adjacent steel and concrete nodes. It can easily be verified that the strain-







Figure 12. Bond Link

displacement relations

$$\epsilon_{x_{1}} = \overline{U}_{3} - \overline{U}_{1}$$

$$\epsilon_{y_{1}} = \overline{U}_{4} - \overline{U}_{2}$$

$$(2.40)$$

yield a  $\begin{bmatrix} b_b \end{bmatrix}$  matrix identical to Equation (2.33). Because of the conceptual similarity between the steel and bond elements, the remaining steps are analogous to those in section 2.4.4 and need not be repeated.

Scordelis has found in his study of reinforced concrete beams (24) that this type of bond mechanism simulates the interaction between concrete and steel quite accurately. It should be mentioned that the linkage element neglects frictional bond, local stress concentrations along the ribs of deformed bars and dowel action.

Research has shown that the redistribution of compressive stresses at the ribs of bars may cause small tensile stresses in the concrete (37). However, for all standard deformed bars the concrete is capable of sustaining these local disturbances. Therefore, these effects are neglected in the design. Frictional bond may be significant, especially near cracks; but it is again neglected since the coefficient of friction is extremely difficult to predict.

Dowel action is usually significant in the corners of bent or curved reinforcements. This study considers straight bars only. Therefore, a bond link which does not account for this effect is justified.

# 2.4.6 Cracked Concrete Element

It is well known that the tensile strength of concrete is only a fraction of its compressive strength. This rather unpleasant property leads to cleavage failure (tension cracking) at relatively small loads. At any load larger than that which causes the concrete to crack, the reinforcements are called upon to resist the entire tensile force. This type of behavior plays an important role in the nonlinear analysis of reinforced concrete.

In the solution procedure presented here, the influence of a crack on a continuous triangular concrete element is treated in a similar way as proposed by Rashid (23) in 1968. The element is cut in the direction perpendicular to the principal tensile stress  $\sigma_1$ . In this new state the element no longer has any stiffness normal to the crack surface (Figure 13).

Consequently, the concrete may be considered as a uniaxial stress condition parallel to the second principal axis. This assumption results in the following stress-strain relationship:

$$\begin{bmatrix} \sigma_1 \\ \sigma_2 \\ 0 \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 \\ 0 & \mathbf{E}_2 & 0 \\ 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} \epsilon_1 \\ \epsilon_2 \\ 0 \end{bmatrix}$$
(2.41)

or, in matrix formulation,

$$\left\{\sigma\right\} = \left[D_{cr}\right] \left\{\epsilon\right\}$$
(2.42)

The stiffness matrix in local (principal) coordinates may now be derived on the basis of Equation (2.29). Hence,

$$\begin{bmatrix} k_{cr, e} \end{bmatrix} = \begin{bmatrix} b \end{bmatrix}^{t} \begin{bmatrix} D_{cr} \end{bmatrix} \begin{bmatrix} b \end{bmatrix} A \qquad (2.43)$$

For the assembly of the total stiffness matrix, the local matrix (Equation (2.19)) must be expressed in terms of global coordinates. Ordinarily, this is accomplished by a matrix triple product of the following form:



Figure 13. Cracked Element

$$\begin{bmatrix} k_{cr, g} \end{bmatrix} = \begin{bmatrix} R \end{bmatrix}^{t} \begin{bmatrix} k_{cr, e} \end{bmatrix} \begin{bmatrix} R \end{bmatrix}$$
(2.44)

where



It should be noted at this point that similar transformations must be performed on all anisotropic, uncracked elements, whose material characteristics follow Equation (2.11).

# CHAPTER III

#### MATRIX ITERATIVE PROCEDURES

# 3.1 <u>Review of Iterative Procedures for Problems with Nonlinear</u> <u>Material Properties</u>

# 3.1.1 General

Nonlinear structures are usually classified according to the cause of nonlinear behavior. Since all solution procedures in solid mechanics involve equilibrium, kinematic and constitutive equations, nonlinearities may arise from either of these three sets of fundamental relations. In case of large displacements, the geometric configuration of the assembly may change sufficiently under load to influence the equilibrium relations. Large deflections also cause nonlinear terms in the kinematic relations. It appears then that nonlinearities may be due to either the geometry or the material properties or both. Thus, the• following three categories contain all possible sources of nonlinear conditions:

1. <u>Geometric nonlinearity</u> caused by nonlinear kinematic relations.

2. <u>Material nonlinearity</u> which arises from complex material laws.

3. <u>Combined</u> geometric and material nonlinearity. The matrix analysis methods developed for linear structures can be extended to include the above mentioned complications. Because of

the presence of nonlinear terms, the solution to the governing matrix equations can no longer be obtained explicitly. Consequently, the use of iterative procedures is inevitable. Most of the early applications handle nonlinearities by calculating corrections to linear solutions. A common method used in the solution of geometrically nonlinear systems is due to Turner et al. (38). The structure is solved as a sequence of elastic problems in which corrective stiffness matrices are generated to update the geometry. A comprehensive review of such methods and subsequently developed procedures can be found in Oden's paper on nonlinear structural analysis (21).

Similar iterative schemes have also been adopted in the study of inelastic structures. Among the earliest applications were investigations concerning thermal effects and creep (39, 5). The most significant developments are connected with research on elasto-plastic problems. Basically, four methods have emerged from such investigations:

- 1. Direct iterative approach;
- 2. Initial strain approach;
- 3. Variable elasticity approach;
- 4. Initial stress approach.

The key to these different methods is the formulation of the matrix of elastic constants [D]. Since the coefficients of these matrices are functions of the state of stress or strain, they must be re-evaluated after each cycle. In the case of a uniaxial state of stress, the modulus of elasticity may be read from the stress-strain curve directly. More generally, under multiaxial stress conditions, the elastic constants will depend on the stress or strain invariants. It is reasonable to assume that the effective stress  $\sigma_{eff}$  is equal to the value of the second

invariant  $\rm J_2$  of the stress deviator tensor. Similarly, the effective strain  $\rm e_{eff}$  corresponds to the second strain deviator invariant  $\rm I_2,$  where:

$$J_{2} = \frac{1}{6} \left\{ (\sigma_{x} - \sigma_{y})^{2} + (\sigma_{y} - \sigma_{z})^{2} + (\sigma_{z} - \sigma_{x})^{2} + 6(\sigma_{xy}^{2} + \sigma_{yz}^{2} + \sigma_{zx}^{2}) \right\}$$
(3.1)  

$$I_{2} = \frac{1}{6} \left\{ (\epsilon_{x} - \epsilon_{y})^{2} + (\epsilon_{y} - \epsilon_{z})^{2} + (\epsilon_{z} - \epsilon_{x})^{2} + 6(\epsilon_{xy}^{2} + \epsilon_{yz}^{2} + \epsilon_{zx}^{2}) \right\}$$
(3.2)

Or, in terms of principal values,

$$J_{2} = \frac{1}{6} \left\{ (\sigma_{1} - \sigma_{2})^{2} + (\sigma_{2} - \sigma_{3})^{2} + (\sigma_{3} - \sigma_{1})^{2} \right\}$$
(3.3)

$$I_{2} = \frac{1}{6} \int (\epsilon_{1} - \epsilon_{2})^{2} + (\epsilon_{2} - \epsilon_{3})^{2} + (\epsilon_{3} - \epsilon_{1})^{2} \Big\}$$
(3.4)

The effective or equivalent stress is introduced for convenience as

$$\sigma_{\text{eff}} = \frac{1}{V_2} \left\{ (\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \right\}^{\frac{1}{2}} \quad (3.5)$$

It is closely related to the frequently used octahedral shear stress

$$\tau_{\text{oct}} = \frac{1}{3} \left\{ (\sigma_1 - \sigma_2)^2 + (\sigma_2 - \sigma_3)^2 + (\sigma_3 - \sigma_1)^2 \right\}^{\frac{1}{2}}$$
(3.6)

through the following relationship:

$$\sigma_{\rm eff} = \frac{3}{V_2} \tau_{\rm oct} \tag{3.7}$$

Most authors prefer to use either one of these quantities in place of the rather abstract term  $J_2$ :

$$\sigma_{\rm eff} = (3J_2)^{\frac{1}{2}}$$
 (3.8)

$$\tau_{\text{oct}} = \left(\frac{2}{3} J_2\right)^{\frac{1}{2}} \tag{3.9}$$

The convenience of the above definitions and the corresponding strain terms becomes apparent in the formulation of yield criteria. For example, the von Mises condition relates the second stress invariant of the multi-dimensional state of stress to the uniaxial case as follows:

$$3J_2 = \sigma_{\text{eff}}^2 = \sigma_0^2$$
 (3.10)

where  $\sigma_0$  is the uniaxial tensile or compressive stress. Thus, the three-dimensional situation may be expressed in terms of one parameter,  $\sigma_0$ , only. Furthermore, if a unique relationship between  $\sigma_0$  and  $\epsilon_0$  exists, one is able to determine the material constants for the three-dimensional continuum at any load level from the uniaxial stress-strain curve:

$$\sigma_{o} = f(\epsilon_{o})$$

$$\sigma_{eff} = f(\epsilon_{eff}) \qquad (3.11)$$

# 3.1.2 Direct Iterative Approach

The direct iterative technique is based upon repeated elastic solutions, where for each cycle the full load is applied. Initially, all elements are assigned a modulus of elasticity,  $E_0$ , and a Poisson's ratio,  $v_0$ , corresponding to zero stress. Subsequently, the elastic constants are redefined for each new solution. They depend on the state of stress (or strain) reached in the previous step. According to Zienkiewicz (12), an adequate solution requires three to four iterations.

Unfortunately, this simple method has several disadvantages. It is, for example, impossible to include an unloading cycle in a problem. Clearly, during a load decrease the plastic strains should remain constant. Since the procedure is entirely based upon total effective strains, a reduction in load may cause a change in plastic deformation. Furthermore, it is difficult to obtain reasonable, consistent representations of the equivalent Poisson's ratio.

Gallagher et al. (39), and Argyris (8) realized that both difficulties can be avoided by incremental procedures. These step-by-step methods have the additional advantage that they permit the use of incremental stress-strain characteristics such as the Prandtl-Reuss equations.

# 3.1.3 Initial Strain Approach

The procedure here consists of applying the load in small increments. For any such load interval the incremental stresses and strains are computed. Total stresses and strains may be obtained by adding the current incremental values to the total stresses (or strains) reached during the previous step. Clearly, the evaluation of the elastic strain increments is straightforward. However, the change in plastic strain depends on both the initial and final stress condition and cannot be determined directly.

The total incremental strain  $\{\Delta e\}$  in any interval may, in general, consist of elastic, plastic, thermal and creep strain increments. Throughout this study the latter two contributions will be neglected. Hence, the total strain increments reduce to

$$\left\{\Delta \mathbf{e}\right\} = \left\{\Delta \boldsymbol{\epsilon}_{\mathbf{e}}\right\} + \left\{\Delta \boldsymbol{\epsilon}_{\mathbf{p}}\right\} \tag{3.12}$$

If the plastic strain increments are known, they may be treated as initial strains  $\{\Delta \epsilon_{0, p}\}$  similar to those caused by temperature changes. Consequently, the stress increments can be determined through a standard elastic analysis.

$$\left\{ \Delta \sigma \right\} = \left[ D \right] \left( \left\{ \Delta \epsilon_{e} \right\} - \left\{ \Delta \epsilon_{o, p} \right\} \right)$$
(3.13)

The difficulites involved in establishing the plastic strain increments depend on the degree of sophistication desired. Any plastic constitutive relation may be implied, including time or temperature dependent material characteristics. One of the most commonly used set of equations, the Prandtl-Reuss flow rule, will be elaborated as an example. In matrix notation they take on the form

$$\left\{\Delta\epsilon_{p}\right\} = C_{p}\left[D_{o}\right]^{-1}\left\{\sigma\right\}$$
(3.14)

where  $C_p$  is a function of the effective stress and the effective plastic strain increment,  $\Delta\varepsilon_{eff}^p$  .

$$\Delta \epsilon_{\text{eff}}^{p} = \frac{\sqrt{3}}{2} \left\{ \left( \Delta \epsilon_{1}^{p} - \Delta \epsilon_{2}^{p} \right)^{2} + \left( \Delta \epsilon_{2}^{p} - \Delta \epsilon_{3}^{p} \right)^{2} + \left( \Delta \epsilon_{3}^{p} - \Delta \epsilon_{1}^{p} \right)^{2} \right\}^{\frac{1}{2}}$$

$$+ \left( \Delta \epsilon_{3}^{p} - \Delta \epsilon_{1}^{p} \right)^{2} \right\}^{\frac{1}{2}}$$

$$(3.15)$$

The matrix  $\left[D_{o}\right]^{-1}$  contains v = 0.5; thus, for the three-dimensional case

$$\begin{bmatrix} D_{o} \end{bmatrix}^{-1} = \frac{1}{E} \begin{bmatrix} 1 & -.5 & -.5 \\ -.5 & 1 & -.5 \\ -.5 & -.5 & 1 \\ -.5 & -.5 & 1 \\ -.5 & -.5 & 1 \\ -.5 & -.5 & 1 \\ -.5 & -.5 \\ -$$

and

$$C_{p} = \frac{\Delta \epsilon_{eff}^{p}}{\sigma_{eff}}$$
(3.17)

To obtain the effective incremental plastic strain, Argyris (8) suggested two different methods. First, the so-called direct incremental approach makes use of results obtained in the preceding step. Assume that upon completion of increment i-1, the total and incremental stresses and strains are available. The values of  $\Delta \epsilon_{eff}^{p}$  and  $\sigma_{eff}$  are readily determined from Equations (3.15) and (3.5), respectively. The modulus of elasticity,  $E_{o}$ , and Poisson's ratio,  $v_{o}$ , for the zero stress condition are used throughout the entire solution. With  $E_{o}$  and  $v_{o}$  known, the constitutive equations needed for the formulation of the stiffness matrices are defined as

$$\left\{\Delta \mathbf{e}\right\}_{i} = \left[\mathbf{D}\right]^{-1} \left\{\Delta\sigma\right\}_{i} + \mathbf{C}_{p,i} \left[\mathbf{D}_{o}\right]_{i}^{-1} \left\{\Delta\sigma\right\}_{i} \qquad (3.18)$$

This procedure may be improved by performing, for each load step, an initial elastic solution and a series of subsequent iterations. The increments of stresses and strains of the current cycle are used to obtain a new estimate of the plastic strain increment for the next iteration. According to Argyris (8), this iterative-incremental method usually converges after five iterations.

It should be mentioned that both methods require special precaution when unloading occurs. During a load decrease, the structure must behave in a purely elastic fashion which may be accomplished by specifying the factor  $C_p$  as zero. Likewise, upon reloading,  $C_p$  must remain zero until the current  $\sigma_{eff}$  is found to exceed the highest effective stress achieved during the previous increment.

# 3.1.4 Variable Elasticity Approach

For elastic-perfectly plastic and ideally plastic material, the methods of sections 3.1.3 and 3.1.4 break down. This is due to the

fact that large plastic strain increments may result even from very small load augmentations. Pope suggested a method in 1965 (7) which adjusts the stress-strain relationship in every load increment to take into account plastic deformations. The works of Marcal and King (11), Akyuz and Merwin (15) fall into the same category.

For the elastic strain increment the expression remains as

$$\left\{\Delta\epsilon_{\rm e}\right\} = \left[{\rm D}_{\rm e}\right]^{-1} \left\{\Delta\sigma\right\} \tag{3.19}$$

However, the Prandtl-Reuss equations (which express the plastic strain increments in terms of actual accumulated stresses  $\{\sigma\}$ ) must be replaced by a relationship of the form

$$\left\{\Delta\epsilon_{\mathbf{p}}\right\} = \left[\mathbf{D}_{\mathbf{p}}\right]^{-1} \left\{\Delta\sigma\right\}$$
(3.20)

To derive  $[D_p]^{-1}$ , let H' denote the slope of the effective stresseffective plastic strain function, which, again, will be assumed to be available through experiments. The strain-hardening criterion in discrete form then becomes

$$\Delta \sigma_{\rm eff} = H' \Delta \epsilon_{\rm eff}^{\rm p} \tag{3.21}$$

By differentiating the von Mises yield condition, a second expression for  $\Delta \sigma_{eff}$  may be obtained as follows:

$$\Delta \sigma_{\text{eff}} = \frac{3}{2\sigma_{\text{eff}}} \left\{ \sigma_{x}^{\dagger} \Delta \sigma_{x} + \sigma_{y}^{\dagger} \Delta \sigma_{y} + \sigma_{z}^{\dagger} \Delta \sigma_{z} \right\}$$
$$+ \frac{3}{\sigma_{\text{eff}}} \left\{ \sigma_{xy} \Delta \sigma_{xy} + \sigma_{yz} \Delta \sigma_{yz} + \sigma_{zx} \Delta \sigma_{zx} \right\}$$
(3.22)

where

$$\sigma'_{x} = \frac{1}{3} (2\sigma_{x} - \sigma_{y} - \sigma_{z})$$
 (cyclic sub-  
stitution) (3.23)

The above term can now be written in matrix form as

$$\Delta \sigma_{\text{eff}} = \{S\}^{t} \{\Delta \sigma\}$$
(3.24)

and substituted into Equation (3.21).

$$\Delta \epsilon_{\text{eff}}^{\text{p}} = \frac{1}{\text{H}'} \left\{ S \right\}^{\text{t}} \left\{ \Delta \sigma \right\}$$
(3.25)

Upon substitution of Equation (3.25) into the Prandtl-Reuss relations (3.14), one arrives at the desired incremental stress-strain rule.

$$\left\{ \Delta \epsilon_{\rm p} \right\} = \frac{\Delta \epsilon_{\rm eff}^{\rm p}}{\sigma_{\rm eff}} \left[ D_{\rm o} \right]^{-1} \left\{ \sigma \right\}$$
$$= \frac{1}{{\rm H}' \sigma_{\rm eff}} \left[ D_{\rm o} \right]^{-1} \left\{ \sigma \right\} \left\{ {\rm S} \right\}^{\rm t} \left\{ \Delta \sigma \right\}$$
(3. 26)

or

$$\left\{\Delta\epsilon_{\mathbf{p}}\right\} = \left[\mathbf{D}_{\mathbf{p}}\right]^{-1} \left\{\Delta\sigma\right\}$$
(3.27)

Hence,  $\begin{bmatrix} D_p \end{bmatrix}$  depends upon the current state of stress  $\{\sigma\}$  and the strain-hardening history through the parameter H'. Combined with the elastic constitutive relations, the total strain increments become

$$\left\{\Delta \mathbf{e}\right\} = \left\{\Delta \boldsymbol{\epsilon}_{\mathbf{e}}\right\} + \left\{\Delta \boldsymbol{\epsilon}_{\mathbf{p}}\right\} = \left(\left[\mathbf{D}_{\mathbf{e}}\right]^{-1} + \left[\mathbf{D}_{\mathbf{p}}\right]^{-1}\right) \left\{\Delta \boldsymbol{\sigma}\right\} (3.28)$$

and the corresponding change in stress is

$$\left\{\Delta\sigma\right\} = \left(\left[D_{e}\right]^{-1} + \left[D_{p}\right]^{-1}\right)^{-1}\left\{\Delta\epsilon\right\}$$
(3.29)

This particular method is known to converge very rapidly. Furthermore, unloading can be treated by simply inserting an elastic  $\begin{bmatrix} D \end{bmatrix}$  matrix in the increment following an unloading interval. From a computational point of view, the variable elasticity approach has one disadvantage; at each solution step, the stiffness of the structure is changed. Thus, for every iteration the whole structural stiffness matrix must be reassembled, which naturally results in excessive computer time.

#### 3.1.5 Initial Stress Method

The most recent method of elasto-plastic analysis was introduced by Zienkiewicz et al. (20) as an alternative approach to the "variable elasticity" procedure. This "initial stress" method makes use of the fact that the total strain increments uniquely define the corresponding stress situation throughout the entire load history. This holds true for any type of stress-strain relationship including those for ideally plastic structures. Therefore, it seems more reasonable to treat the stress increments as initial values rather than the strains. The change in stress derived from the corresponding strain increment will, in general, be incorrect. Consequently, the initial stress approach must again rely on an iteration scheme.

Once more, the first step in each load increment consists of solving the problem elastically. Both the strain increments  $\{\Delta \epsilon_e\}$  and the associated change in stress  $\{\Delta \sigma_e\}$  are computed. Since the calculated values for  $\{\Delta \sigma_e\}$  deviate from the true stress increments  $\{\Delta \sigma\}$ , the equilibrium conditions are violated. In order to maintain equilibrium, a set of "body forces" equal and opposite to the initial stress system  $\{\Delta \sigma_b\}$  must be introduced.

$$\left\{\Delta\sigma_{\rm b}\right\} = \left\{\Delta\sigma_{\rm e}\right\} - \left\{\Delta\sigma\right\} \tag{3.30}$$

In the computation, the unknown, true stress increments are replaced by approximative values  $\left\{\Delta \overline{\sigma}\right\}$  determined from the second iteration cycle.

Before proceeding to the second solution step, each element is examined for its type of behavior. For this purpose the calculated stress increments are added to the total stresses  $\{\sigma_0\}$  reached during the preceding increment to establish the current stresses  $\{\sigma\}$ .

$$\left\{\sigma\right\} = \left\{\sigma_{o}\right\} + \left\{\Delta\sigma_{e}\right\}$$
(3.31)

These values and the corresponding strain-hardening parameter k are substituted into a suitable yield criterion,  $F(\{\sigma\}, k)$ . The resulting numerical value determines whether the element exhibits elastic or plastic behavior. From the theory of plasticity, it is known that for a strain-hardening material, the following four cases may be distinguished:

a.F < 0elastic behaviorb.F = 0 and  $\Delta F < 0$ unloading, elastic behaviorc.F = 0 and  $\Delta F = 0$ neutral loading, plastic behaviord.F = 0 and  $\Delta F > 0$ loading, plastic behavior

where

$$\Delta \mathbf{F} = \frac{\partial \mathbf{F}}{\partial \{\sigma\}} \left\{ \Delta \sigma \right\}$$
(3.33)

Zienkiewicz states the same conditions in a more computer-oriented form (29).

Clearly, no further iteration is required if, after the beginning elastic cycle, the first or second condition is satisfied throughout the entire structure. Otherwise, the solution is continued by computing new stress increments  $\left\{\Delta\overline{\sigma}\right\}$ .

$$\left\{\Delta\overline{\sigma}\right\} = \left[D_{ep}\right]\left\{\Delta\epsilon_{e}\right\}$$
(3.34)

where  $\begin{bmatrix} D_{ep} \end{bmatrix}$ , the matrix of material constants, is a rather involved expression. Its coefficients are dependent upon the yield condition, the strain-hardening parameter and the stress-strain curve. In the case of elasto-perfectly-plastic material, the matrix still exists since the slope of the stress-strain curve H' appears as a single term in the numerator (20).

All subsequent iterations must now be based on augmented load conditions (initial, externally applied load increments plus equilibrating nodal forces). For the ith solution cycle, they are

$$\left\{\overline{\mathbf{P}}_{i}\right\} = \left[\mathbf{b}\right]^{t} \left\{\Delta \sigma_{i}^{*}\right\} dV \qquad (3.35)$$

where

$$\left\{ \Delta \sigma^{*} \right\} = \left\{ \Delta \overline{\sigma}_{i} \right\} - \left\{ \Delta \overline{\sigma}_{i-1} \right\}$$
(3.36)

# 3.2 Proposed Iteration Procedure

The method presented here is based upon an iterative, incremental load approach. For each load increment, the whole structure is repeatedly solved as an elastic problem until closure. Consider an arbitrary concrete element during load increment i. Assume that at the end of the previous step the principal strains  $\{e_{p, i-1}\}$  and stresses  $\{\sigma_{p, i-1}\}$  have been established. Based upon these values the element may be in any one of the following conditions:

- a. Type 1: Elastic, isotropic;
- b. Type 2: Elastic, anisotropic;
- c. Type 3: Inelastic, anisotropic;
- d. Type 4: Cracked.

The four cases can be visualized diagrammatically in Figures 14 and 15.

In the present computer program, principal strains  $\{e_{i-1}\}$  are used to determine the relevant material constants. After the proper modulus of elasticity,  $E_i$ , and Poisson's ratio,  $v_1$ , have been found for



(a) TYPE 1



# (b) TYPE 2

Figure 14. Classification of Concrete Elements in the Elastic Range



(a) TYPE 3



# (b) TYPE 4



each element, the  $\begin{bmatrix} D \end{bmatrix}$  matrices are generated. To recapitulate, the appropriate equation shall be summarized below:

Type 1:

$$\begin{bmatrix} \mathbf{D} \end{bmatrix} = \frac{\mathbf{E}_{1}}{(1 - \nu^{2})} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1 - \nu}{2} \end{bmatrix}$$
(3.37)

Types 2 and 3:

$$\begin{bmatrix} D_{a} \end{bmatrix} = \frac{E_{1}}{1 - v_{12}v_{21}} \begin{bmatrix} 1 & v_{12} & 0 \\ v_{12} & n & 0 \\ 0 & 0 & 0 \end{bmatrix}$$
(3.38)

(for principal axes)

Type 4:

$$\begin{bmatrix} \mathbf{D}_{cr} \end{bmatrix} = \mathbf{E}_2 \begin{bmatrix} 0 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$
(3.39)

(for principal axes)

The elemental stiffness matrices  $\begin{bmatrix} k_i \end{bmatrix}$  follow immediately as  $\begin{bmatrix} k_i \end{bmatrix} = \begin{bmatrix} b \end{bmatrix}^t \begin{bmatrix} D \end{bmatrix} \begin{bmatrix} b \end{bmatrix}$  (3.40)

or, for cases 2, 3, and 4,

$$\begin{bmatrix} k_i \end{bmatrix} = \begin{bmatrix} R^t \end{bmatrix} \begin{bmatrix} b \end{bmatrix}^t \begin{bmatrix} D_j \end{bmatrix} \begin{bmatrix} b \end{bmatrix} \begin{bmatrix} R \end{bmatrix}$$
(3.41)

Next, the total stiffness matrix is assembled and solved for the incremental displacements  $\{\Delta U\}$ . The discussion of the procedure used will

be postponed until Chapter IV. The incremental strains are now evaluated as

$$\left\{\Delta e_{i}\right\} = \left[b_{i}\right] \left\{\Delta U_{i}\right\}$$
(3.42)

and added to the total strains  $\{e_{i-1}\}$  of the preceding step to give the new total strains

$$\left\{ e_{i}^{1} \right\} = \left\{ e_{i-1} \right\} + \left\{ \Delta e_{i} \right\}$$

$$(3.43)$$

These values constitute a new strain situation with a corresponding new set of principal strains  $\{e_{p,i}\}$ . The material properties of the following iteration cycle are again extrapolated from the stress-strain curve. The iteration is stopped after a specified tolerance is reached. Before proceeding to the next increment, all total stress and strain values are updated and stored. Similar treatment is imposed upon the reinforcements and bond links. However, the procedure here is much less involved since the matrix [D] reduces to a single term  $E_1$ .

# CHAPTER IV

# COMPUTER PROGRAM

# 4.1 General

The iteration procedure described in Chapter III has been programmed for solution on a digital computer. Two programs, NARCOS-1 and NARCOS-2,<sup>1</sup> were written for the IBM 360-65 model operated by the Oklahoma State University Computer Center. The standard ASA FORTRAN language was used.

Both programs generate all necessary mesh data from a minimum of input information. NARCOS-1 was based on the element arrangement used by Scordelis (24). This first version had to be abandoned at an early stage because of unsatisfactory results. A convergence study revealed that the symmetric mesh of NARCOS-2 converges much more rapidly, Figure 16.

#### 4.2 Computer Idealization of the Beam

The finite element representation of the beam is arranged into rectangles. Each rectangular unit is subdivided into four triangular elements of equal area. The corners are numbered in a clockwise direction, Figure 17. To ensure small band widths, the joint numbers

<sup>&</sup>lt;sup>1</sup>NARCOS is the abbreviation for "Numerical Analysis of Reinforced Concrete Structures."



Figure 16. Results of Convergence Study

. :. \*



(a) ELEMENT AND NODAL ARRANGEMENT





Y



(c) CUMULATIVE NODAL ARRANGEMENT


are arranged in the direction of the least number of nodes. Reinforcements can be connected at the corner joints of the rectangles only. Inclined reinforcing (such as shear reinforcements under 45° wrt. beam axis) is not allowed. The program connects all steel nodes by means of bond links automatically. As mentioned earlier, this requires a revision of the nodal list. All input information must be specified in terms of the original nodal arrangement, Figure 17a. At the end of the data input, the program generates a cumulative nodal list which includes the additional steel nodes, Figure 17b.

# 4.3 Flow Chart

Figure 18 shows a summary flow chart of the program NARCOS-2, using symbols shown in Figure 19. The detailed listing is given in Appendix B.

#### 4.4 Solution of Equilibrium Equations

With the nodal arrangement discussed above, the stiffness matrix for the finite element assembly will be banded. This type of matrix lends itself well to direct solution by Gauss elimination. Since the matrix is symmetric, only the upper half of the band is stored in a rectangular array. The assembly and solution of this array is done blockwise, Figure 20. The first step consists of a forward elimination. Each reduced block is stored on a disk. With the last reduced block still in core, the backward elimination is performed in reverse order. A lthough several load vectors may be included in this procedure, only one loading case is considered in the present program.







Figure 18. General Flow Chart (continued)



PROCESS







SUBROUTINE

START OF MAIN

PROGRAM OR

ARITHMETIC COMPUTATION

DO LOOP CONTROL WITH CORRESPONDING CONNECTOR

UNCONDITIONAL OR CONDITIONAL CONTROL WITH CONNECTOR(S)

CONDITIONAL CONTROL



INPUT

OUTPUT



OUTPUT

MAGNETIC DISK MANIPULATIONS

CONNECTOR



END, RETURN



OR STOP



Figure 20. Block Arrangement of Main Stiffness Matrix

The solution of the stiffness matrix is done in subroutine BANSOL. A detailed description of the standard Gauss procedure shall be omitted. Complete information on this method may be found in Reference 40.

# 4.5 Input List

The input data are arranged in tabular form. Topological and geometrical properties make up the first block. The second block of information consists of the material properties. Block 3 contains the list of reinforcements. Loads and boundary conditions are specified in the last block. Each table is identified by a block header card as shown in Figure 21. The specific format of each type of data statement is given in Appendix C. In the following the general input sequence will be described in detail:

1. <u>Number of problems</u>: The first card must specify the number of problems to be solved.

2. <u>Problem title</u>: One alphanumeric card initializes and identifies a new problem.

3. <u>Control card for first block</u>: Topology and geometry. NAR-COS-2 offers three modes for the input of geometric and topological properties. The mode is specified, together with the number of nodes in the horizontal and vertical direction and the number of reinforcements, on the first control card.

Mode 1. Equal spacing:

Under this mode all rectangular elements are of equal size. The program divides the length and depth of the beam into a specified number of intervals, respectively.



Figure 21. Arrangement of Input Data

The geometry may in this case be given by the coordinates of the upper right corner (for example, node 32 in Figure 17b).

Mode 2. Unequal spacing:

This input option allows for the variation of the element size in horizontal and/ or vertical direction. Example 1 of Appendix D demonstrates this input mode.

Mode 3. Individual input:

Mode 3 requires the declaration of the geometry and topology (i.e., node and element numbers) for each triangular element. This option was included to make the program available for irregular shapes.

4. Joint coordinates: Data giving the location of either the corner node (mode 1) or of nodes which specify the interval length of unequally spaced elements (mode 2) follow the first header card. Input under mode 2 may best be explained by means of Example 1 in Appendix D. The coordinates of the interval points (1, 3, 9) in y-direction (cards 4 to 6) are given first. The number of the joints must correspond to the actual node number (Figure 32). The computer divides the intervals into the correct number of equal segments.

5. <u>Control card for second block</u>: Material properties. This data statement initializes the input list of the stress-strain characteristics and specifies the number of elements with irregular thickness.

6. <u>Element thickness</u>: The program assumes a unit thickness over the entire continuum. However, specification of other than unit thicknesses of the rectangular elements is permitted.

7. <u>Material properties</u>: The stress-strain curves for concrete, steel and bond are given by a set of points on the curve. For steel, the maximum number of stations is not to exceed 20. The stress-strain characteristics for concrete and bond are given by a maximum of ten points. The point, stress equal to zero and strain equal to zero, must also be specified (if the curve passes through the origin). Following the input of the stress-strain graphs, the Poisson's ratios for each region between two points on the curves must be declared. The input of Poisson's ratios is required for concrete and steel only. The last card in this block contains the bond link stiffness,  $K_v$ . This coefficient expresses the strain-relative displacement characteristic of the linkage element perpendicular to the reinforcement. If the numerical value is omitted (blank card), the program assumes a number which is  $10^6$  times the value of the first coefficient on the diagonal of the main stiffness matrix.

8. <u>Reinforcement information</u>: There is no special header card for this table, since the number of reinforcement cards was given in the control statement for the first block. Each reinforcement input specifies one or more bars. The cross-sectional area may be given directly or in the standard form as bar number according to the ACI Code. The position of reinforcements is specified by the number of the start and end node. The computer automatically divides the bar into two-force members of the same length as the corresponding rectangular concrete elements and assigns the proper bond links.

9. <u>Tolerance</u>: The number on this card declares the percentage error tolerated on the largest displacement.

10. <u>Control card of last block</u>: Loading and boundary conditions. The number of loads, boundary conditions, increments and iterations are specified. If the number of increments is equal to zero, the program performs an elastic solution without iterations. Automatic scaling is done if the parameter NINCR is equal to 1. In this case, the load is taken as total load. During the first cycle, the problem is solved elastically and all results are scaled until the element strains in the extreme element correspond to the yield values. The loads are adjusted accordingly. The difference of the total given load and the load at yield is divided into 20 increments.

11. Loading information: One card per joint load, i.e., x- or y- component or both, must be supplied.

12. <u>Boundary conditions</u>: Only one specific boundary condition may be stated on one card. The direction is identified in alphanumeric form (x or y). Prescribed displacements may be introduced by simply adding the numerical value of the induced delfection after the direction parameters, x and y.

To complete this section, a few additional remarks concerning the data input seem necessary. It should be mentioned that the entire input for a particular problem must be consistent with regard to dimensions. The program does not allow for mixed units.

When no reinforcements are specified, several portions of the input sequence are skipped. In this case, the user must omit stressstrain curves for steel and bond, "perpendicular" bond stiffness, and reinforcement cards.

For problems with fixed boundary nodes, it is not necessary to restate these boundary conditions for each load increment. If the program encounters a blank card after the input of a load increment, the reading of boundary conditions is suppressed. The conditions of the first (initial) load step are assumed to apply throughout the entire loading history.

A special feature has been introduced in the program in the form of a CHECK EQUILIBRIUM card. If such a card is included at the end of the first set of boundary conditions, the program checks the equilibrium for each vertical strip of elements. The shear equilibrium and bending stress equilibrium (including forces in steel tendons) are checked. The average shear and normal stresses for each rectangular unit and the residual force on the cross section is output at the end of each load increment.

## 4.6 Output Information

The complete list of input data is printed in tabular form. The calculated topological and geometrical properties refer to the updated cumulative nodal list. Results are provided after each successfully completed load increment. The results consist of a complete list of nodal displacements and stresses and strains in all elements. Several supplementary messages are included to clarify the large output. In addition, two "print error" subroutines report the most common input mistakes. A few are automatically corrected. The corrections made are recorded as nonfatal error messages. Sample output is included in Appendix D.

#### CHAPTER V

#### NUMERICAL RESULTS

# 5.1 General

Three singly reinforced concrete beams were solved with NAR-COS-2. All examples were chosen to be simply supported and acted upon by concentrated loads as shown in Figures 22, 23, 27 and 30.

Because of the symmetric loading, boundary and geometric conditions, the solution could be performed for half beams only. The output consisted of stresses in the concrete and reinforcements together with the principal stresses and their directions for each triangular element. Bond forces as well as the relative displacements between steel and concrete nodes were printed. Several indicators, such as the condition of the concrete elements (e.g., cracked) and equilibrium checks, were included in the listing to facilitate the interpretation of the extensive computer output. Selected results will be discussed in the following sections.

# 5.2 Example Problem 1: Scordelis' Beam A-1

Several purely elastic problems were considered to serve as a check on the computational procedure and the program development. Existing programs for the analysis of in-plane loaded plates were primarily used for this purpose.

Next, Scordelis' beam A-1 was solved with NARCOS-2. The results reported in his paper (24) could be considered as a general guide only because of several differences in the problem setup. First, artificial cracks were introduced in beam A-1 at the beginning of the loading process. The procedure here was to let cracks develop in the direction perpendicular to the principal tensile stresses when the maximum allowable tensile stress was reached. Second, Scordelis did not allow for any tensile resistance in the concrete. In this study, the modulus of rupture was assumed to be 300 psi. As a result of this, the beam under investigation showed a higher loading capacity before cracking took place. Thus, Scordelis' initial load of 1000 pounds had to be modified to  $P_0 = 7000$  pounds. Cracking was initiated at a load level of 7200 pounds. Several cracks developed simultaneously at the bottom edge of the beam. Obviously, this type of crack pattern was to be expected with the moment being constant between the support and the point of application of the load (support at midspan). Subsequently, the load was increased by 200 pound increments up to 8800 pounds. The cracks continued to develop in the same direction (i.e., perpendicular to the edge) and additional cracks occurred along the bottom edge. The stress patterns, including the bond forces and crack propagation, are shown for different load levels in Figures 24 through 26. Figure 22 contains the information about the stress-strain characteristics used. Poisson's ratios of 0.3 for the reinforcements and of 0.15 for the concrete were chosen to be constant throughout the entire load The values for the bond stiffness corresponded to those given history. by Scordelis ( $K_h = 2.2 \times 10^6$  psi).







NUMBER OF TRIANG. EL	EMENTS	NTEL	= 8	864
NUMBER OF NODES (IN	CL. STEEL)	NUMNOD	= 4	195
NUMBER OF LOADS		NLOAD	=	I
NUMBER OF BOUND. CON	ND.	NBOUND	=	10

r

Figure 23. Mathematical Model of Beam A-1







Figure 25. Example Problem 1: Stresses at P = 8000 Lbs.





As mentioned earlier, the introduction of new nodes for the reinforcement required the renumbering of all nodes. To illustrate the difference between the intitial mesh and modified nodal arrangement, the two numbering systems are shown in Appendix D accompanied by the listing of the data input statements.

# 5.3 Example Problem 2: Simple Beam Loaded at Midspan

A simply supported beam acted upon by a concentrated load at midspan was chosen to study the crack propagation in the concrete, Figure 27. This type of structure and load configuration seemed particularly suited for such an investigation, because relatively large cracks should be expected to develop near the center of the beam. An initial load of P = 1500 pounds was applied. The cracks appeared during the first load increment of  $\Delta P = 100$  lbs (i.e., at P = 1600 lbs). The cracking moment obtained from the simple beam theory was found to correspond to a load of  $P_c = 1564$  lbs.

Figures 28 and 29 illustrate the crack pattern at different load levels. The cracked elements have been identified by shading. Small vertical cracks first appear in the bottom elements of the beam. During the next few load increments, new cracks occur in the elements above those which have already cracked. In addition, small vertical cracks appear along the bottom edge farther away from the load. For higher load levels, the cracked elements tend to group in the vicinity of the center of the beam. Near the bottom edge the cracks remain practically vertical. However, Figure 29 clearly indicates that the directions of the cracks in the higher elements begin to point toward the load. Local disturbances in the crack pattern may be observed in



(a) BEAM 2



(b) MATERIAL PROPERTIES

Figure 27. Example Problem 2: Simple Beam Loaded at Midspan









the neighborhood of the supports, especially the two triangular elements, six and seven, exhibit no cracks at all but experience relatively high compressive stresses in the x-direction. This irregularity may be caused by the vertical load at node four (the total concentrated load has been distributed over the nodes at the supports). Also, the steel reinforcement is fixed in the x-direction at support joint three. Thus, the applied vertical load at joint four must be transferred to the steel through the two concrete elements in question, causing high compressive stresses in these elements.

Several other elements at the bottom edge of the beam did not crack because of local stress redistributions. Element 124, for example, is obviously situated between two cracks. There the tensile stresses seem to have decreased enough in order not to cause cracking. Farther out in the beam the crack distribution becomes more regular with uncracked elements occurring more frequently.

# 5.4 Example Problem 3: Simple Beam Loaded Symmetrically by Two Concentrated Loads

A third, simple beam problem was solved to study the nonlinear stress distribution in the compressive zone of the concrete beam after cracking has taken place. The same structural model as in Problem 2 was loaded symmetrically by two concentrated loads at 30 inches from midspan, Figure 30.

Cracks developed again at the bottom edge and continued to extend vertically. The stress-distribution for only one load level will be reported to demonstrate the stress distribution in the compression zone. Due to the small number of relatively large elements in this problem, the stress distribution must be regarded as a crude approxi-



# FOR MATERIAL PROPERTIES, SEE FIGURE 27.

Figure 30. Example Problem 3: Simple Beam Loaded Symmetrically by Two Concentrated Loads

mation. However, the results clearly show the nonlinear character of the concrete stress block above the neutral axis, Figure 31.

.





#### CHAPTER VI

# SUMMARY AND CONCLUSIONS

# 6.1 Summary

The feasibility of the finite element method in the investigation of reinforced concrete beams composed of Hookean material has been established by Scordelis (24). The objective of this thesis was to evaluate the potential of the finite element approach in the study of the nonlinear behavior of reinforced concrete beams under static loads and to provide a tool for the investigation of such structures.

The actual situation was approximated by a structural model of the plane stress type using a finite number of triangular, constantstrain, concrete elements, linear steel bars, and a mechanism to simulate the interaction between the two materials. In selecting the proper material constants for the concrete elements, the principal stresses were used to determine whether an element behaves isotropically, anisotropically or develops a crack. All material characteristics were replaced by piecewise linear stress-strain curves. The finite element approach was implemented in the form of a combined step-bystep, iterative procedure. Three example problems were solved on a digital computer.

#### 6.2 Conclusions and Recommendations

The proposed finite element, step-by-step, iterative procedure is a feasible method to analyze reinforced, concrete beams. The simulation of the inelastic behavior by quasi-anisotropic, finite elements has shown satisfactory results. Likewise, the bond link models appear to approximate the bond slip phenomenon quite accurately. Two major difficulties had to be overcome in connection with the computer solution. First, the introduction of additional steel nodes presented some assembly problems. The use of a cumulative nodal list for the assemblage of such mathematical models proved to be extremely helpful. This technique allows the introduction of any number of additional nodes within any basic nodal configuration. Second, some difficulties were experienced with the bond link stiffness coefficients. To suppress relative displacements between steel and concrete nodes perpendicular to the steel bars, large numerical values for the vertical stiffness coefficients had to be used. These values must be selected with great caution. Extremely large numbers may cause completely erroneous solutions.

The convergence of the iterative process has been found to be slow for large load increments. On the other hand, too many small increments will result in excessive computer time. It would be advantageous to employ the solution procedure proposed by Zienkiewicz (20) which does not require the assemblage of the structural stiffness matrix for each iteration.

The present method may be recommended for extension to include time dependent effects such as creep or initial stress conditions resulting from temperature or prestressing forces. The method could be

modified for three-dimensional assemblies. However, the need for larger digital machines becomes even more apparent for such models.

There is considerable doubt that the method could be used for dynamic loading conditions. Additional iteration cycles would probably increase the computer time tremendously. In addition, the program would have to be modified to include the possibility of unloading conditions.

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# APPENDIX A

# STIFFNESS MATRICES IN TABLE FORM

#### A.1 Isotropic Stiffness Matrix for Triangular Concrete Panels

Using the numbering system shown in Figure 10 and the straindisplacement transformation matrix from Equation (2.32), the following stiffness matrix results from Equation (2.35). For convenience, the matrix is separated into two component matrices (Equations (A.1) and (A.2)):  $\begin{bmatrix} K_s \end{bmatrix}$  represents the stiffness due to shear and  $\begin{bmatrix} K \end{bmatrix}$  contains terms due to normal stresses only.

#### A.2 Anisotropic Stiffness Matrix for Triangular Concrete Panels

The stiffness matrix (Equation (A.3) below) is in terms of local coordinates, the axes being u and v, Figure 7. Again, the nodes are numbered in clockwise directions. The matrix becomes much less complex for the principal axes, since  $\overline{G}$  vanishes.

#### A.3 Stiffness Matrix of Steel Bars

Linear bar stiffness matrices are conveniently given in global coordinates directly, i.e., after the transformation (Equation 2.41) has been performed.

$$\begin{bmatrix} K_{g} \end{bmatrix} g = \frac{EA}{L} \begin{bmatrix} \frac{c_{x}^{2}}{c_{x}} & \frac{c_{y}^{2}}{c_{y}} & \frac{c_{y}^{2}}{c_{y}} & \frac{c_{y}^{2}}{c_{x}} & \frac{c_{y}^{2}}{c_{x}} & \frac{c_{y}^{2}}{c_{x}} & \frac{c_{y}^{2}}{c_{x}} & \frac{c_{y}^{2}}{c_{x}} & \frac{c_{y}^{2}}{c_{y}} & \frac{c_{y}^{2}}{c_{y}} \end{bmatrix}$$
(A.6)

where

$$\begin{bmatrix} K \end{bmatrix} = \frac{Et}{4A(1 - \nu^2)} \begin{bmatrix} y_{32}^2 & & & \\ -\nu y_{32} x_{32} & x_{32}^2 & & \\ -y_{32} y_{31} & \nu x_{32} y_{31} & y_{31}^2 & & \\ \nu y_{32} x_{31} & -x_{32} x_{31} & -\nu y_{31} x_{31} & x_{31}^2 & & \\ y_{32} y_{21} & -\nu x_{32} y_{21} & -\nu x_{31} y_{21} & \nu x_{31} y_{21}^2 & & \\ -\nu y_{32} x_{21} & x_{32} x_{21} & \nu y_{31} x_{21} & -x_{31} x_{21} & -\nu y_{21} x_{21}^2 \end{bmatrix}$$
(A.1)

$$\begin{bmatrix} K_{s} \end{bmatrix} = \frac{Et}{8A(1+\nu)} \begin{bmatrix} x_{32}^{2} & & & \\ -x_{32}y_{32} & y_{32}^{2} & & \\ -x_{32}x_{31} & y_{32}x_{31} & x_{31}^{2} & & \\ x_{32}y_{31} & -y_{32}y_{31} & -x_{31}y_{31} & y_{31}^{2} & & \\ x_{32}x_{21} & -y_{32}x_{21} & -x_{31}x_{21} & y_{31}x_{21} & x_{21}^{2} & \\ -x_{32}y_{21} & y_{32}y_{21} & x_{31}y_{21} & -y_{31}y_{21} & -x_{21}y_{21} & y_{21}^{2} \end{bmatrix}$$

(A.2
	$\int \frac{2}{y_{00}} +$				
	x <sub>32</sub> <sup>2</sup> G				
	$-x_{32}y_{32}v_{21}$ $nx_{32}^{2}$		•	•	
	-x <sub>32</sub> y <sub>32</sub> G +y <sub>32</sub> <sup>2</sup> G				
	<sup>-y</sup> <sub>32</sub> <sup>y</sup> <sub>31</sub> <sup>x</sup> <sub>32</sub> <sup>y</sup> <sub>31</sub> <sup>ν</sup> <sub>21</sub>	y <sub>31</sub> <sup>2</sup>			
$\begin{bmatrix} K \end{bmatrix} = \frac{E_1 t}{2E_1 t}$	-x <sub>32</sub> x <sub>31</sub> G +x <sub>31</sub> y <sub>32</sub> G	+x <sub>31</sub> <sup>2</sup> G		· · ·	
$\begin{bmatrix} -a \end{bmatrix} 4A(1 - \nu_{12}\nu_{21})$	$x_{31}y_{32}v_{21}$ $-x_{31}x_{32}n$	-x <sub>31</sub> y <sub>31</sub> <sup>v</sup> 21	x <sub>31</sub> <sup>2</sup> n		
• • • • •	+x <sub>32</sub> y <sub>31</sub> G -y <sub>31</sub> y <sub>32</sub> G	-x <sub>31</sub> y <sub>31</sub> G	+y <sub>31</sub> <sup>2</sup> G		
	$y_{21}y_{32}$ $-x_{32}y_{21}v_{21}$	-y <sub>21</sub> y <sub>31</sub>	$x_{31}y_{21}v_{21}$	y <sub>21</sub> <sup>2</sup>	
	$+ x_{21} x_{32} \overline{G} - x_{21} y_{32} \overline{G}$	-x <sub>21</sub> x <sub>31</sub> G	+x <sub>21</sub> y <sub>31</sub> G	+x <sub>21</sub> <sup>2</sup> G	
	$-x_{21}y_{32}v_{21}$ $x_{32}x_{21}^{n}$	*21 <sup>y</sup> 31 <sup>v</sup> 21	<sup>-x</sup> 21 <sup>x</sup> 31 <sup>n</sup> 21	-x <sub>21</sub> y <sub>21</sub> <sup>ν</sup> 21	x21 <sup>n</sup>
	-x <sub>32</sub> y <sub>21</sub> G +y <sub>32</sub> y <sub>21</sub> G	+x <sub>31</sub> y <sub>21</sub> G	-y <sub>21</sub> y <sub>31</sub> G	-x <sub>21</sub> y <sub>21</sub> G	+y21 <sup>2</sup> G
• •	· · · · · · · · · · · · · · · · · · ·				(A.3)

where

$$G = \frac{\overline{G}_{21}}{\overline{E}_1} (1 - \nu_{12} \nu_{21}) \text{ and } G_{21} = \left\{ \left[ \frac{\overline{E}_1}{2(1 + \nu_{21})} \right] \left[ \frac{\overline{E}_2}{2(1 + \nu_{12})} \right] \right\}^{\frac{1}{2}}$$
(A.4)  
$$n = \frac{\overline{E}_2}{\overline{E}_1}$$
(A.5)

86

(A.5)

$$c_x = \cos\theta$$
 (A.7)

$$c_{y} = \sin\theta \qquad (A.8)$$

### A.4 Bond Link Stiffness Matrix

Similar to the "two-force" members, the bond link matrices are given in the datum system directly.

$$\begin{bmatrix} K_{b} \end{bmatrix} g = \begin{bmatrix} \frac{c_{x}^{2}k_{1} + s_{x}^{2}k_{2}}{s_{x}c_{x}k_{1} - s_{x}c_{x}k_{2}} & \frac{s_{x}^{2}k_{1} + c_{x}^{2}k_{2}}{s_{x}c_{x}k_{1} - s_{x}c_{x}k_{2}} & \frac{s_{x}c_{x}k_{1} + s_{x}c_{x}k_{2}}{c_{x}c_{x}k_{1} - s_{x}c_{x}k_{2}} & \frac{s_{x}c_{x}k_{1} + s_{x}c_{x}k_{2}}{c_{x}c_{x}k_{1} + s_{x}c_{x}k_{2}} & \frac{s_{x}c_{x}k_{1} + s_{x}c_{x}k_{2$$

#### A.5 Cracked Concrete Element

	0						
	0					symme- tric	
[K ]=	0	x <sub>32</sub> <sup>2</sup>	0				(A.10)
[_u]	0	<sup>-x</sup> 32 <sup>x</sup> 31	0				
	0	Q	0		0		
	0	<sup>x</sup> 21 <sup>x</sup> 32	0	-x <sub>21</sub> y <sub>31</sub>	0	2 x <sub>21</sub>	

#### APPENDIX B

## LISTING OF COMPUTER PROGRAM

.

	C . A NEW LOAD INCREMENT
•	C .
PROGRAM NARCOS .	C . ITOP(NUMBAR,4) = TOPOLOGY OF REINFORCEMENT BARS
NONLINEAR ANALYSIS OF REINFORCED CONCRETE STRUCTURES	C   C CONTAINS NODE NUMBERS FOR REINFORCEMENTS
•	C • ISTYPE(NUMBAR) = TYPE OF BEHAVIOR OF STEEL ELEMENTS
LANGUAGE USED : FORTRAN IV	C • SES(20,2) = STRESS - STRAIN VECTUR FOR STELL
DIGITAL MACHINE : IBM 360-65 .	C . ES(19) = MODULI OF ELASTICITY FOR STEEL
PROGRAMMER : ALEXANDER J. LASSKER .	C . XNUS(19) = POISSON'S RATIUS FOR STEEL
DATE OF COMPLETION : JUNE 30, 1971 .	C . SAREA(NUMBAR) = STEEL AREA
•	C . ESPS (NUMBAR) = STRAINS IN STEEL REINFORCEMENTS
DESCRIPTION OF PROGRAM .	C . STRSINUMBARI = STRESSES IN STEEL REINFORCEMENTS
•	<b>C</b> •
HIS PROGRAM SOLVES REINFORCED CONCRETE STRUCTURES OF THE	C . A(NH,MA) = WORKING AREA FOR MAIN STIFFNESS MATRIX
LANE STRESS TYPE. THE FINITE ELEMENT METHOD IS USED IN AN .	C . BUNH) = CORRESPUNDING WORKING SPACE FOR LOADS
TERATIVE PROCEDURE. AT EACH LUAD STEP THE PROBLEM IS SOLVED .	C •
S AN ELASTIC PROBLEM. DETAILED INFORMATION CAN BE FOUND IN: .	C . CURRENT SIZE OF PROGRAM
ONLINEAR ANALYSIS OF REINFORCED CONCRETE BEAMS UNDER STATIC .	C •
QADS. PH.D. DISSERTATION BY A.J. LASSKER, QSU, AUGUST 1971.	C NN = 54
•	C • NH = 108
	C MA = 54 (MBAND IS NAX. 54)
•	C • NUMNOD = 620
PROBLEM PARAMETERS USED IN THIS PROGRAM	C • NREL = 288
•	C . NTEL = 1152
N = BLOCK LENGTH .	C .NUMBAR = 50
A = BLOCK WIDTH •	C . MAXNDF = 1300
H = LENGTH OF WORKING AREA .	C .
REL = NUMBER OF RECTANGULAR ELEMENTS .	C . STRESS - STRAIN CURVE GIVEN BY NO MORE THAN 10 POINTS
TEL = NUMBER OF TRIANGULAR ELEMENTS .	C . MODULI OF ELASTICITY GIVEN BY NO MORE THAN 9 POINTS
ODV = NUMBER OF NODES VERTICALLY	C . POISSON'S RATIOS GIVEN BY NO MORE THAN 9 POINTS
ODH = NUMBER OF NODES HORIZONTALLY	C . NUMBER OF LOADS GIVEN IS LESS THAN 5
OF = NUMBER OF DEGREES OF FREEDOM AT ONE NODE	C . NUMBER OF BOUNDARY CONDITIONS GIVEN IS LESS THAN 5
•	
(NUMNOD) = X - COORDINATES .	Č
(NUMNOD) = Y - COORDINATES .	c
HETA(NTEL) = ANGLE OF PRINCIPAL AXES	IMPLICIT REAL +8(A-H.D-7)
TOP(NREL.4) = TOPOLOGY OF CUNCRETE ELEMENTS .	REAL*8 DATAN2 DSIN. DCDS DABS
CONTRINS NODE NUMBERS IN CLOCKWISE DIRECTION	COMMON S( 10-10) . VKH . VKV
HICK(NREL) = THICKNESS OF RECTANGULAR ELEMETS	COMMON DXX . DYY . E1. E2. CNU. CNU12. CNU21.EP1.EP2. SNU. PT
TYPE(NTEL) = TYPE OF BEHAVIOR OF CONCRETE ELEMENTS	COMMON NN + NH + MA + NODV + NODH + NUMPRO- NR EL - NT EL - NIMBAR - MAXNOE, MBANO
EC(10.2) = STRESS - STRAIN VECTOR FOR CONCRETE	COMMON NINCR + KINCR + NG + R 40 - NG + R 45 - NZG + NZS + I SCALE + NDE - LAUTO
CONTAINS STRESSES IN COLUMN 1	COMMON NUMDE NUMT DEAN IT . IT . NCIRYBANZBANZD 2
CONTAINS STRAINS IN COLUMN 2	
S(9) = MODULI OF ELASTICITY FOR CONCRETE	CONMON/ELEM / X(650) • Y(650) • THE TA(1152.2) • U(1300) • TU(1300)
NUC(9) = POISSON'S RATIOS FOR CONCRETE	COMMON/FLEM1 / JTOP (288.4). THICK (288.)
PS(NTEL.3) = STRAINS FOR TRIANGULAR ELEMENTS	
DRINTEL 2) = PRINCIPAL STRAINS FOR TRIANGULAR FLEMENTS	COMMON/TYP / ITYPE(1152), ISTYPE(50), INTYPE(100) KOTP
	COMMON/MODULC/ SECTION21.ECTOR: YNUCTOR: TO
TROINEL, 2) = ODINCIDAL STRESSES FOR TRIANGULAR FLEMENTS	
TRANILLYEY - FRANCIAL STRESSES FOR INTRODUCTN LEGENTS •	
LUAU - HUNDER OF LUAUING VARUS * *	
DES - NUMBER OF DUDNDART CUNDITIONS SPECIFIED (A)T DIR ( ) .	COMMON/ DADSI/ ALONDE 201211 ABOUNDE 20121
CONTRUCTOR - LUNDING VEGTOR	COMMONICANDST ALCANT 2012/14000001 20121
CONTAINS TO CONDUCT A COLUMN 2 $\bullet$	COMMONICENEST ICONST 2017 ISUNICE 2017 ISUNICES
CONTAINS A - CONCORENT IN COLUMN 2 .	COMMONYREINE / SAREAL DUJJURLIDDUJZJJILUPL DUJOJNKEINEJ SIJ
CUNIAINS T - CUMPUNENT IN COLUMN 3 .	COMPONIZONSTRY SIR(1152,3), STRY(1152,2), EPS(1152,3), EPR(1152,2)
BUUNDINBUS,3J = PRESCRIBED DISPLACEMENT VELIUR .	
(NUMNUJENDEJ = NUDAL DISPLACEMENTS FUR A LUAD INCREMENT .	CUMMUN/SILSIK/ ESPSI 501,51KS( 501,TESPS( 50),TSTRS( 50,2)
U(NUMNOD*NDF) = TOTAL NUDAL DISPLACEMENTS	CUMPUN/BUND / EPSB(100,2), STRB(100,2), TEP 58(100,2), TSTRB(100,2)
B(NUMNOD*NDF) = AUXILIARY DISPLACEMENT VECTOR .	UATA UVKV/1.00+06/ , ZER0/0.00+00/

```
EQUIVALENCE (KA(1),K1), (KA(2),K2), (KA(3),K3), (KA(4),K4)
      EQUIVALENCE (KA(5), K5), (KA(6), K6), (KA(7), K7), (KA(8), K8)
      EQUIVALENCE (KA(9), K9), (KA(10), K10)
С
C
      READ NUMBER OF PROBLEMS
С
      READ(5.8000) NPROB
 8000 FORMAT(15)
      KPROB = 0
      IF(NPROB.GT.0) GOTO 100
      WRITE(6,9000) NPROB
 9000 FORMAT(1H1,1X, NUMBER OF PROBLEMS MUST BE GREATER THAN ZERO ./. 1X.
     1 NUMBER OF PROBLEMS SPECIFIED = +,15)
      CALL EXIT
C
С
      EACH NEW PROBLEM STARTS AT STATEMENT NO. 100
      INITIALIZE MAIN PRUBLEN PARAMETERS
c
С
  100 CONTINUE
      KPROB = KPROB + 1
С
      CALL INITL
C
      ICHECK = 0
      CALL READIN(ICHECK)
С
      CALL OUTPUT
С
С
      EACH NEW LOAD INCREMENT STARTS AT STATEMENT NO. 200
С
  2CO CONTINUE
      0 = TI
      DO 210 L = 1,NUMTOF
  210 U(L) = ZERO
ſ
C.
      EACH ITERATION CYCLE STARTS AT STATEMENT NO. 300
С
  300 CONTINUE
      IT
           = IT + 1
      I
             = 0
      15
             = 0
      NUMBLK = 0
      KSHIFT = 0
      ISWICH = 1
      IL
             = 1
      DO 305 IA = 1,NH
      B(IA) = ZERO
      DO 305 JA = 1.MA
  305 A(IA, JA) = ZERU
      DO 310 KU = 1.NUMTOF
  310 UB(KU) = TU(KU) + U(KU)
C
      REWIND 1
      REWIND 2
C
С
      SETUP BLOCKS OF THE MAIN STIFFNESS MATRIX
      EACH BLOCK SETUP STARTS AT STATEMENT NO. 400
£
r
  400 CONTINUE
```

```
NUMBLK = NUMBLK + 1
С
С
      PROCESS ONE ELEMENT AT THE TIME
C
       MTYP = 1
  410 I = I + 1
      J = 4 + I - 3
      JI = JTOP(I_{+}I)
      J2 = JTOP(I,2)
      J3 = JTOP(1,3)
      J4 = JTOP(1,4)
      J5 = J1 + NODV
C
      K2 = 2* JCNL (J1.2)
      K1 = K2 - 1
      IF((K1 - KSHIFT) - NN) 430-430-420
  420 I = I - 1
      GOTO 490
  430 K4 = 2*JCNL (J2.2)
      K3 = K4 - 1
      K6 = 2+JCNL(J3,2)
      K5 = K6 - 1
      K8 = 2 + JCNL(J4,2)
      K7 = K8 - 1
      K10 = 2*JCNL(J5,2)
      K9 = K10 - 1
С
      CALL STIFF(1,MTYP, J, I, J1, J2, J5, 1, 2, 5)
      J = J + 1
      CALL ST IFF (2, MTYP, J, I, J2, J3, J5, 2, 3, 5)
      J = J + 1
      CALL STIFF(2,MTYP, J, I, J5, J3, J4, 5, 3, 4)
      J = J + 1
      CALL STIFF(2, MTYP, J, I, J1, J5, J4, 1, 5, 4)
C
      CHOSE PROPER VKV - VALUE
С
С
  470 IF(NREINF.EQ.0) GOTO 475
С
С
      ASSEMBLY OF RECTANGULAR ELEMENT STIFFNESS MATRIX
C
      FORM BLOCKS OF MAIN STIFFNESS MATRIX
      SIZE OF WORKING AREA IS MA*NH
С
      ONE BLOCK IS HALF OF THE WORKING AREA
C
  475 D0 480 KI = 1.10
      IINA = KA(KI) - KSHIFT
      KINA = KA(KI) - K1 + 1
      DO 480 \text{ KJ} = 1.10
      JS = KA(KJ) - K1 + 1
      IF(JS.LT.KINA) GOTO 480
      JINA = KA{KJ} - KA{KI} + 1
      A(IINA, JINA) = A(IINA, JINA) + S(KI, KJ)
  480 CONTINUE
      IF((VKV.EQ.0.0).AND.(I.EQ.1)) VKV = DVKV*A(1,1)
С
С
      END OF LOOP FOR ELEMENT PROCESSING WITHIN UNE BLOCK
      CHECK IF THE ELEMENT BELONGS TO THE CURRENT BLOCK
C
C
      IF(I.LT.NREL) GUTO 410
```

```
IF ( NREINF .EQ. 0 ) GD TO 560
C
С
      ASSEMBLY OF LINEAR BAR STIFFNESS MATRICES
С
      DO 500 IR = 1.10
      D0 500 JR = 1,10
  500 S(IR, JR) = 0.0
      00 550 IS = 1,NUMBAR
      MTYP = 2
      ISWICH = 1
      1 \text{LINK} = 0
      IS1 = ITOP(IS,3)
      IS2 = ITOP(IS,4)
      K2 = 2*ITOP(IS,5)
      K1 = K2 - 1
      K4 = 2 \times ITOP(IS,6)
      K3 = K4 - 1
      IF ( IS2 .LT. ( IS1 + NOD2 ) ) ISWICH = 2
C
С
      CHECK IF BAR BELONGS TO CURRENT BLOCK
С
      IF((K1.LE.KSHIFT).OR.(K1.GT.(KSHIFT+NN))) GOTO 541
      CALL STIFF(ISWICH, MTYP, IS, IS, IS1, IS2, IS2, 1, 1, 1)
C
      ASSEMBLE
С
C.
  535 DO 540 KI = 1,4
      IINA = KA(KI) - KSHIFT
      KINA = KA(KI) - KI + 1
      00 540 KJ = 1,4
      JS = KA(KJ) - K1 + 1
      IF(JS.LT.KINA) GOTO 540
      JINA = KA(KJ) - KA(KI) + 1
      A(IINA_{2}JINA) = A(IINA_{2}JINA) + S(KI_{2}KJ)
  540 CONTINUE
  541 ILINK = ILINK + 1
      GOTO (545,548,550) , ILINK
c
      CHECK IF LINK BELONGS TO CURRENT BLOCK
C
      IF IT DOES NOT, SKIP BOTH LINKS
С
c.
  545 \text{ Kll} = 2 \text{ JCNL}(IS1,2) - 1
      IF((K11.LE.KSHIFT).OR.(K11.GT.(KSHIFT+NN))) GOTO 548
С
      BOND LINK FOR START NODE OF STEEL BAR
C
С
  546 K44 = K4
      K33 = K3
      K4 = K2
      K3 = K1
      K1 = K11
      K2 = K11 + 1
      IV = IS
      GOT 0 549
C
      BOND LINK FOR END NODE OF STEEL BAR
С
```

```
548 IF ( IS .NE. NUMBAR ) GO TO 550
```

490 CONTINUE

```
K2 = 2 \neq JCNL (152.2)
      K1 = K2 - 1
      IF(K1.LE.KSHIFT.OR.K1.GT.(KSHIFT+NN)) GOTU 550
      K4 = K44
      K3 = K33
      IV = IS + 1
  549 CONTINUE
      II = 2 = IS - ISWICH + 1
      EP1 = TEPSB(II,1)
      MTYP = 3
      CALL STIFF ( ISWICH, MTYP, IV, IS, IS, IS, I, 1, 1)
      GOTO 535
  550 CONTINUE
C
      PROCESS LOADS AND BOUNDARY CONDITIONS FOR EACH BLOCK
C
      PUT CONCENTRATED LOADS FROM ARRAY XLOAD INTO B
C
C
  560 DO 580 IL = 1, NLOAD
      JL = 2+ JCNL(ILDAD(IL),2) - KSHIFT
      IF(((JL-1).GT.NN).OR.((JL-1).LE.O)) GOTO 570
      B(JL-1) = XLOAD(1L,1)
  570 IF((JL.GT.NN).OR.(JL.LE.O)) GOTO 580
      B(JL) = XLOAD(IL,2)
  580 CONTINUE
С
C
      PROCESS BOUNDARY CONDITIONS
      MODIFY EQUATIONS FOR SPECIFIED DISPLACEMENTS AT BOUNDARY
С
С
      MODIFICATIONS FOR STEEL BOUNDARY CONDITIONS INCLUDED
C
      DO 600 IB = 1, NBCS
      IF( IBOUND( IB).LE. 0) GOTO 590
      JB = 2*JCNL(IBOUND(IB), 2) - KSHIFT
      IF(((JB-1).GT.NH).OR. ((JB-1).LE.O)) GOTO 600
      CALL MODIFY((JB-1), XBOUND(IB, 1))
      IF(JCNL(IBOUND(I8),1).EQ.0) GUTO 600
      IF(((JB+1).GT.NH).OR.((JB+1).LE.OJ) GOTO 600
      CALL MODIFY ((J8+1), XBOUND(IB, 1))
      GOTO 600
  590 JBN = -IBOUND(IB)
      JB = 2+JCNL (JBN, 2) - KSHIFT
      IF((JB.GT.NH).OR.(J8.LE.0)) GOTO 600
      CALL MODIFY(JB, XBOUND(IB, 2))
      IF(JCNL(JBN,1).EQ.0) GUTO 600
      IF(((JB+2).GT.NH).OR.((JB+2).LE.0)) GOTO 600
      CALL MODIFY((JB+2), XBOUND(IB,2))
  600 CONTINUE
C
      WRITE BLOCK ON TAPE 2 AND SHIFT LOWER PART INTO UPPER PART
¢
C.
      WRITE(2) (B(N), (A(N, M), M=1, MBAND), N=1, NN)
      DU 610 N = 1.NN
      K = N + NN
      B(N) = B(K)
      B(K) = 0.0
      DO 610 M = 1, MBAND
      A(N \cdot M) = A(K \cdot M)
  610 A(K,M) = 0.0
      KSHIFT = KSHIFT + NN
c
```

```
C
      CHECK FOR LAST BLOCK
С
      COMPLETE LOADS AND BOUNDARY CONDITIONS IN LAST BLOCK
C.
      IF I = NREL , INCREASE NUMBLE AND I BY 1
C
      IF(I - NREL) 400,620,630
  620 IF((2+JCNL(JTOP(1.3).2)).LE.KSHIFT) GOTO 630
      NUMBLK = NUMBLK + 1
            ______
                    + 1
      1
      GOTG 560
C
С
      SOLVE SYSTEM OF EQUATIONS BLOCKWISE BY GAUSS ELIMINATION
С
  630 CONTINUE
С
      CALL BANSOL
С
      KU = 0
      NIN = NN + 1
      D0 640 N8U = 1, NUMBLK
      DO 640 NMU = NIN, NH
      KU = KU + 1
  640 U(KU) = A(NHU, NBU)
С
      CALCULATE STRAINS, PRINCIPAL STRAINS, STRESSES AND PRINCIPAL STR.
C
C
      DO 670 I = 1. NR EL
      J = 4 + I - 3
      J1 = JTOP(1,1)
      J2 = JTOP(1,2)
      J3 = JTOP(1,3)
      14 = JTOP(1,4)
      J5 = J1 + NODV
      HTYP = 1
      CALL STRESS(1,J,J1,J2,J5,HTYP)
      J = J + 1
      CALL STRESS(1, J, J2, J3, J5, MTYP)
      1 = 1 + 1
      CALL STRESS (1, J, J5, J3, J4, MTYP)
      J = J + 1
  670 CALL STRESS (1, J, J1, J5, J4, MTYP)
      IF(NREINF.EQ.0) GOTO 690
      MTYP = 2
      DO 680 J = 1, NUMBAR
      JS1 = 1TOP(1,3)
      JS2 = ITOP(J,4)
  680 CALL STRESS (J, J, JS1, JS2, JS2, MTYP)
С
      DETERMINE I TERATION PROCEDURE
С
      SEARCH FOR LARGEST ERROR IN DISPLACEMENTS
С
С
  690 IFI(NINCR.EQ.0).OR. (KINCR.EQ.1)) GOTO 720
      1F(IT.EQ.1) GOTU 300
      DIFF = ZERO
      LTOL = 1
      DO 700 L = 1, NUMT DF
      DIFF1 = DABS((TU(L) + U(L)) - UB(L))
      IF(DIFF1.LE.DIFF) GOTO 700
      DIFF = DIFF1
      LTOL # L
```

```
700 CONTINUE
      DTOL = DABS(TOL*(TU(LTOL) + U(LTOL)))
      WRITE(6.9020) DTOL.LTOL
 9020 FORMAT(/,1X, TOLERANCE = *,1PD12.4,* AT DISPLACEMENT*,15)
      IF(DIFF.LE.DTOL) GOTO 720
  710 IF(IT.LT.NIT) GOTO 300
      WRITE(6,9100) NIT, DIFF, DTOL
 9100 FORMAT(7,1x, "ERROR IN I TERATION NO ",13," IS BIGGER THAN TOLERANCE
     1',/,1X,'ERROR = ',1PD12.5,3X,'TOLERANCE = ',1PD12.5,/,1X,'NUMBER G
     2F I TERATIONS INCREASED BY 5 ./ )
      NIT = NIT + 5
      IF(NIT.LT.15) GOTO 300
      WRITE(6,9200)
 9200 FORMAT (////.1x. SOLUTION STOPPED BECAUSE NIT EXEEDS 15)
      CALL EXIT
С
      CALCULATE TOTAL STRAINS, STRESSES AND DISPLACEMENTS
C
C.
  720 DO 800 KU = 1,NUMTDF
  800 TU(KU) = TU(KU) + U(KU)
      DO 810 KE = 1, NTEL
      DO 810 JE = 1,3
      TEPS(KE, JE) = TEPS(KE, JE) + EPS(KE, JE)
  810 TSTR(KE, JE) = TSTR(KE, JE) + STR(KE, JE)
      IF (NREINF.EQ.0) GOTO 8020
      DO 815 KS = 1,NUHBAR
      TESPSIKS) = TESPSIKS) + ESPSIKS)
  815 TSTRS(KS,1) = TSTRS(KS,1) + STRS(KS)
      NB11 = 2 + ( NUMBAR + 1 )
      DO 816 JS = 1,2
      DO 816 KS = 1. NB11
      TEPSB(KS, JS) = TEPSB(KS, JS) + EPSB(KS, JS)
  816 TSTRB(KS,JS) = TSTRB(KS,JS) + STRB(KS,JS)
 8020 CALL RESOUT (ICHECK)
С
      IF(NINCR.EQ.0) GUTU 850
      IF(NINCR.GT.1) GOTO 830
      IF(ISCALE) 830,820,830
C
С
      SCALE LOADS AND PRESCRIBED DISPLACEMENTS
С
  820 ISCALE = 1
      CALL SCALE
      GOTO 200
C.
      PROCESS NEW LOAD INCREMENT
С
c
  830 IF(NINCR - KINCR) 460,850,840
  840 CALL SCALE2
      GOTG 200
  850 WRITE(6,9010) KPROB
 9010 FORMAT(//,1X,'END OF PROBLEM',15)
      IF(KPROB.LT.NPROB) GOTO 100
  460 CALL EXIT
      END
      SUBROUTINE INITL
C
C
C.
```

DO LOOPS ARE MORE EFFICIENT THAN DATA STATEMENTS С С • С • PARAMETERS USED: С . MUMNOD = NUMNOD С . MREL = NREL c . MTEL = NTEL c . MUMBAR = NUMBAR £ MLOAD = NLOADR С . = NBCS C MBCS . С . С С IMPLICIT REAL+8 (A-H, D-Z) COMMON S(10,10), VKH, VKV COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI COMMON NN, NH, MA, NODV, NODH, NUMNOD, NR EL, NT EL, NUMBAR, MAXNDF, MBAND COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO COMMON NUMDE, NUMTOF, NIT, IT, NCURVE, NZ8, NOD2 C COMMON/ELEN / X(650), Y(650), THE TA(1152,2), U(1300), TU(1300) COMMON/ ELEM1 / JTOP (288,4), THICK (288) COMMON/POOL / A(108,54),B(108),NUMBLK COMMON/TYP / ITYPE(1152), ISTYPE(50), IBTYPE(100), KOIR COMMON/MODULC/ SEC(10,2), EC(9), XNUC(9), TOL COMMON/MODULS/ SES(20,2), ES(19), XNUS(19) COMMON/MODULB/ SEB(10,2), EB(9) COMMON/LOADS / XLOAD( 20,2), XBOUNO( 20,2) COMMON/LOADS1/ ALOAD( 20,2),ABOUND( 20,2) COMMON/LOADS2/ ILOAD( 20), IBOUND( 20), NLOAD, NBC S COMMON/REINF / SAREA( 50), JCNL(650, 2), ITOP( 50, 6), NREINF, IS1, IS2 COMMON/CUNSTR/ STR(1152,3), STRP(1152,2), EPS(1152,3), EPR(1152,2) COMMON/CONST1/ TSTR(1152,3), TEPS(1152,3) COMMON/STLSTR/ ESPS( 50),STRS( 50),TESPS( 50),TSTRS( 50,2) COMMON/BOND / EPSB(100,2), STRB(100,2), TEPSB(100,2), TSTRB(100,2) C = 3.1415926535898 ΡI NDF \* 2 MUNTON = 650 MUMNOD = 650 = 288 MREL = 4 \* MREL ATEL MUMBAR = 50 MLOAD = 20 MBCS Ŧ 20 c KINCR × I SCALE = ۵ IAUTO . 0 **ITYPES** E 1 KTYP £ 1 ITYP = 1 С MA з 54 NN . 54 NH = NN + NN NUMTOF = MUMTON\*NDF С

THIS SUBROUTINE INITIALIZES ALL ARRAYS USED IN THE PROGRAM

С

DO 10 I = 1.MUMNOD JCNL(I,1) . 0 JCNL(1,2) = 1 = 0.0 X{[] 10 Y(I) . 0.0 DO 20 I = 1, MREL THICK(I) = 1.0 DO 20 J = 1.420 JTOP(I.J) 0 × DD 30 I = 1,NUMTOF TU(I) 0.0 -30 U(I) 0.0 DO 50 I = 1,MTEL ITYPE(I) \* 1 D0'40 J = 1,3EPS(I,J) = 0.0 TEPS(1,J) = 0.0 TSTR(I,J) 0.0 \* STR(I,J) 40 . 0, 0 0050 J = 1.2THETA(I,J) = 0.0 EPR(I,J) = 0.0 50 STRP(1,J) = 0.0 DD 60 I = 1.NH 8(1) ≠` 0**.**0 DO 60 J = 1, MA6 C A(I,J) = 0.0 DO 80 I = 1.NUMBAR ISTYPE(I) = IBTYPE(1) -1 IBTYPE(I+MUMBAR) = 1 SAREA(I) x 0.0 ESPS(I) = · 0.0 STRS (1) = 0.0 TESPS(I) **z** ( 0.0 TSTRS(I,1) = 0.0 TSTRS(1,2) . 0.0 DO = 80 J = 1.680 ITOP(I,J) **=** 0 NUMBO = 2\*NUMBAR DO 70 I = 1,NUMBD D0 70 J = 1.2EPS8(I,J) = 0.0 STRB(1,J) = 0.0 TEPSB(1,J) = 0.0 70 TSTRB(I,J) -0.0 DG 90 I = 1.MLOAD ILOAD(I) = 0 00.90 J = 1.2ALDAD(1,J) ⇒ 0.0 90 XLOAO(I,J) = 0.0 DO 100 I = 1, MBCS IBOUND(I) **≖** ∩  $00 \ 100 \ J = 1.2$ XBOUND(I,J) = 0.0 100 ABOUND(I,J) = 0.0 00 110 I = 1.9EC(1) = 0.0 110 XNUC(I) = 0.0

С.

```
DG 120 I = 1.10
      120 = 1_{20}
      SEB(I,J)
                       = 0.0
  120
        SEC(I,J)
                       .
                         0.0
      DO 130 I = 1,19
        ES(I)
                       * 0.0
  130
        XNUS(I)
                         0.0
      DS 140 I = 1,20
     00 140 J = 1,2
 140
        SES(I,J)
                       * 0.0
С
      RETURN
      END
      SUBROUTINE READIN(ICHECK)
c
C
    _____
C
    . SUBROUTINE READIN READS ALL INPUT INFORMATION AND DOES ALL
С
    . NECESSARY AUTOMATIC NUMBERING PROCESSES
C
С
    C
      INPLICIT REAL+8(A-H,0-Z)
     COMMON S(10,10), VKH, VKV
      COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI
      COMMON NN, NH, MA, NODV, NODH, NUMNOD, NR EL, NT EL, NUMBAR, MAXNDF, MBAND
     COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO
      COMMON NUMDE, NUMT DE, NIT, IT, NCUR VB, N28, NOD2
С
      COMMON/ELEM / X(650),Y(650),THETA(1152,2),U(1300),TU(1300)
      COMMON/ELEM1 / JTOP (288,4), THICK (288)
      COMMON/MODULC/ SEC(10,2),EC(9),XNUC(9),TOL
      COMMON/MODULS/ SES( 20,2) , ES( 19) , XNUS( 19)
      COMMON/MODUL B/ SEB(10,2),EB(9)
      COMMON/LOADS / XLOADI 20,21,XBOUNDI 20,21
     COMMON/LOADS1/ ALDAD( 20,2),ABOUND( 20,2)
      COMMON/LOADS2/ ILOADE 20), IBOUNDE 20), NLOAD, NBCS
     COMMON/REINE / SAREA( 50), JCNL(650,2), ITOP( 50,6), NREINF, IS1, IS2
      DATA D20/1.00-20/ , ZERO/0.0D 00/ , LABEL1/*X*/ , LABEL2/*Y*/
      DATA IEQUI/*EQUI*/
     DIMENSION NAME(18)
C.
8000 FORMAT(18A4)
 8010 FORMAT( 41 5)
 8020 FORMAT(515)
 8030 FORMAT(15,2012.4)
 8040 FORMAT(15,012.4)
 8050 FORMAT (15, D12.4, D12.6)
 8060 FORMAT(15,012.6)
 8070 FORMAT(15.2012.4)
 8080 FORMAT(15,4X,A1,D12.4)
 8090 FURMAT(215,012.4,215)
8100 FORMAT(5X,D12.5)
 8110 FORMAT(A4)
 90C0 FORMAT(1H1,//,1X,79(1H*),/,2H *,77X,1H*,/,1X,1H*,3X,18A4,2X,1H*,/,
     12H *,77X,1H*,/,2H *,78(1H*),//)
9010 FORMAT(//,1X, ** FINITE ELEMENT ANALYSIS OF */ * PLANE STRESS REINF
     10RCED CONCRETE STRUCTURES! /)
9020 FORMAT (1X, ** PROGRAM CHECKS EQUILIBRIUM*/)
```

 $\mathbf{i}$ 

```
9030 FORMAT(//)
     READ(5,8000) (NAME(I),I=1,18)
     WRITE(6,9000) (NAME(I), I=1,18)
     READ NODAL INFORMATION
     READ (5,8010) NNV, NNH, KT OP, NREINF
     IF(NREINF.LT.0) CALL PRERI(6)
     NOOV = 1ABS (NNV)
     NODH = IABS(NNH)
     NELV = NODV - 1
     NELH = NODH - 1
     IF(KTOP.EQ.1) GOTO 30
     NREL = NELV*NELH
     ATEL = 4*NREL
     NUMNOD = NODV*NODH + NREL
     NOD2 = 2 = 2 = 1000 = 1
     INDIVIDUEL INPUT OF TOPOLOGY
     DO 10 N = 1, NREL
   10 READ(5,8020) IP,(JTOP(IP,J),J=1,4)
     DG 20 N = 1,NUMNOD
   20 READ(5,8030) K,X(K),Y(K)
     READ(5.8010) MBAND
     GOTO 220
      AUTOMATIC PROCESSING OF NODAL ARRANGEMENTS
  30 READ(5,8040) JJ, OTEMP
     IF(NNV) 40,2000,60
      EQUAL SPACING
   40 DD = DTEMP/NELV
      DDD = DD/2.0
      DO 50 N = 1,NELV
      Y(N) = (N - 1) + DD
   50 Y(NODV + N) = Y(N) + DDD
      Y(NODV) = NELV+DD
      GUTO 90
      UNEQUAL SPACING
C
   60 L = 1
      D = 0.0
      DO 80 I = 1,NELV
      READ(5,8040) J. DYTEMP
      K = J - JJ
      DD = (DYTEMP - DTEMP)/K
      DO 70 LK = 1.K
      Y(L) = D
      L = L + 1
   70 D = D + DD
      DTEMP = DYTEMP
   80 JJ ≖ J
      Y(L) = D
      NODV = JJ
```

C

С

С

c

c

C.

С

£

C

С

С

c

£

£

```
NELV = NODV - 1
      NOD2 = 2*NODV - 1
      DO 85 LV = 1,NELV
   85 Y(LV + NODV) = 0.5+(Y(LV+1) - Y(LV)) + Y(LV)
   90 READ(5,8040) MM, DTEMP
      D = 0.
      [F(NNH) 100,2010,130
C
      EQUAL SPACING
С
C
  100 D = DTEMP/NELH
      CDCD = D/2.0
      NI = 1
      D0 120 N = 1, NELH
      X(NI) = (N - 1) = 0
C
      UNEQUAL SPACING
C
      DO 110 NV = 1, NELV
      X(NI + NV) = X(NI)
  110 X(NI + NV + NELV) = X(NI) + DDDD
  120 NI = NI + 2*NODV - 1
      X(NI) = X(NI-1) + DDDD
      DO 125 NV = 1, NELV
  125 X(NI + NV) = X(NI)
      GOTO 170
С
  130 KSTART = 1
      KEND = NOOV
      00 160 I = 1,NELH
      READ(5,8040) M. DXTEMP
      K = A - MM
      DD = (DXTEMP - DTEMP)/K
      DC = DD/2.0
      00 150 LK = 1.K
      DO 140 L = KSTART, KEND
  140 \times LI = D
      IF(LK.EQ.1) GOTO 145
      DO 142 KS = 1.NELV
  142 X(KSTART - KS) = X(KSTART) - DC
  145 KSTART = KSTART + NOD2
      KEND = KSTART + NELV
   150 D = D + DD
      NOD4 = NOD2 - 1
      KD = KEND - NOD2
      DO 155 KS = 1,NOD4
  155 X(KEND - KS) = X(KD - KS) + DD
      DTEMP = DXTEMP
  160 MM = M
      X(KEND) = X(KEND - 1)
      NODH = MM
      NELH = MM - 1
  170 NREL = NELV*NELH
      NTEL = 4*NREL
      NUMNOD = NOD V+NODH + NREL
      NOD2 = 2 \neq NDDV - 1
      NUMDE = NUMNOD*NDE
      NSTART = 0
      I = 1
  160 DO 190 K = 1,NELV
```

```
MK = NSTART + K
      NKK = NK + NUD2
      JTOP(1,1) = NK
      JTOP(1,2) = NK + 1
      JTOP(1,3) = NKK + 1
      JTOP(1,4) = NKK
  190 I = I + 1
      IF(I.GT.NREL) GOTO 200
      NSTART = NSTART + NOD2
      GŪ TO 180
C
      CHECK NUMBERING
2000 IPRER = 2
  959 CALL PRERI(IPRER)
2010 \text{ IPRER} = 1
      GOTC 999
 2020 IPRER = 8
      GOT 0 999
  200 CONTINUE
      NOD3 = NOD2 + 1
      DO 210 KNS = NOD3, NUMNDD
  210 Y(KNS) = Y(KNS - NOD2)
  220 CONTINUE
С
С
      READ STRESS - STRAIN LAWS
ċ
      READ(5,8010) NTH, NCURVC, NCURVS, NCURVB
      IFINTH.EQ.01 GOTO 240
      READ(5, 8040) NT, THIC
      IF(NT.GT.0) GOTO 211
      DO 21 N = 1, NREL
   21 THICK(N) = THIC
      GOTO 240
  211 IF(NT.EQ.1) GOTO 212
      DO 230 N = 2,NTH
  230 READ(5.8040) NT.THICK (NT)
      GO TO 240
  212 THICK(1) = THIC
  240 CONTINUE
c
      CONCRETE
С
c
      DO 250 N = 1,NCURVC
      READ(5, 8050) K, SEC(K, 1), SEC(K, 2)
  250 IF((SEC(N,1).EQ.0.0).AND.(SEC(N,2).EQ.0.0)) NZC = N
      NNC = NCURVC - 1
      DO 260 N = 1, NNC
      EC(N) = (SEC(N+1,1) - SEC(N,1))/(SEC(N+1,2) - SEC(N,2))
  260 READ(5,8060) K, XNUC(K)
      IF(NREINF.EQ.0) GOTO 490
C
      STEEL
С
C.
      DO 270 N = 1.NCURVS
      READ(5,8050) K, SES(K,1), SES(K,2)
  270 IF((SES(N,1).EQ.0.0).AND.(SES(N,2).E4.0.0)) NZS = N
      NNS = NCURVS - 1
      DO 280 N = 1, NNS
      ES(N) = (SES(N+1,1) - SES(N,1))/(SES(N+1,2) - SES(N,2))
```

280 READ(5,8060) K,XNUS(K)

```
BOND
c
С
      00 290 N = 1, NCURV8
      READ(5,8050) K, SEB(K,1), SEB(K,2)
  290 IF((SEB(N,1).EQ.0.0).AND.(SEB(N,2).EQ.0.0)) NZB = N
      NNB = NCURVB - 1
      DO 300 N = 1.NNB
 300 EB(N) = (SEB(N+1,1) - SEB(N,1))/(SEB(N+1,2) - SEB(N,2))
      READ(5.8100) VKV
ſ
      READ INFORMATION ABOUT REINFORCEMENT
£
c
      NUMBAR = 0
      DO 350 NJC = 1,NUMNOD
  350 \text{ JCNL}(NJC_{2}) = 0
      DO 450 NR = 1, NREINF
      READ(5.8090) NBAR.NO.RAREA.NBEG.NEND
      IF(NEND - NBEG) 370,360,380
  360 \text{ KPRER1} = 3
      CALL PRERI (KPRER1)
  370 NNBEG = NBEG
      NBEG = NEND
      NEND = NNBEG
      KPRER2 = 5
      CALL PRER2(KPRER2)
  380 IF(DABS(RAREA).LT.D20) RAREA = ZERO
      IF(RAREA) 390,400.410
  390 \text{ KPRER1} = 4
      CALL PRERI (KPRERI)
  400 RAREA = NBAR*(0.25*PI*(NO+0.125)*(NO+0.125))
С
      CHECK WHETHER HORIZONTAL OR VERTICAL BAR
  410 KEND = NBEG + NOD2
      IF(NEND.GE.KEND) GOTO 430
      VERTICAL BAR
C
  420 NUMBAR = NUMBAR + 1
      ITOP(NUMBAR, 1) = NBAR-
      ITOP(NUMBAR, 2) = ND
      ITOP(NUMBAR,3) = NBEG
      ITOP(NUMBAR, 4) = NBEG + 1
      JCNL(NBEG_{1}) = 1
      JCNL(NBEG + 1,1) = 1
      SAREA(NUMBAR) = RAREA
      NBEG = NBEG + 1
      IF((NBEG+1) - NEND) 420,420,450
C.
      HOR IZONTAL BAR
  430 NUMBAR = NUMBAR + 1
      ITOP(NUMBAR, 1) = NBAR
      ITOP (NUMBAR, 2) = NO
      ITOP(NUMBAR,3) = NBEG
      ITOP (NUMBAR. 4) * KEND
      JCNL(NBEG_{2}) = 1
      JCNL(KEND_2) = 1
      SAREA(NUMBAR) = RAREA
      NBEG = KEND
      KEND = NBEG + NOD2
      IF(NEND.GE.KEND) GUTO 430
      IF(((NEND+NOD2) - KEND).EQ.0) GOTD 450
      KPRER1 = 5
```

C.

```
CALL PRERI(KPRERI)
      GO TO 430
  450 CONTINUE
      DU 460 N # 1.NUMNOD
  460 \text{ JCNL}(N, 1) = \text{ JCNL}(N, 1) + \text{ JCNL}(N, 2)
c
       GENERATE CUMULATIVE NODE LIST
С
ċ
       JCNL(1,2) = 1
      DU 470 N = 2, NUMNOD
  470 \text{ JCNL}(N_{2}) = \text{ JCNL}(N-1_{2}) + \text{ JCNL}(N-1_{2}) + 1
       DU 480 IS = 1, NUMBAR
       IBOND = 1
       JBOND = 1
       IS1 = ITOP(IS,3)
      IS2 = ITOP(IS,4)
      KS1 = JCNL(IS1,2)
       KS2 = JCNL(IS2,2)
      IF((JCNL(IS1,1).EQ.2).AND.(ITOP(IS,4).LT.(ITOP(IS,3) + NOD2)))
     1 \text{ IBGND} = 2
      IF((JCNL(IS2,1).EQ.2).AND.(ITOP(IS,4).LT.(ITOP(IS,3) + NOD2)))
      1J80ND = 2
      ITOP(ISF5) = KS1 + IBOND
  480 \text{ ITOP(IS,6)} = \text{KS2} + \text{JBOND}
С
  490 CONTINUE
C
       READ TOLERANCE
C.
С
       READ(5,8100) TOL
C
       READ LOADING INFORMATION AND BOUNDARY CONDITIONS
С
Ċ.
       READ(5, BO10) NLOAD, NBCS, NINCR, NIT
       DU 310 N = 1.NLOAD
       READ(5,8070) ILOAD(N),XLOAD(N,1),XLOAD(N,2)
       ALOAD(N, 1) = XLOAD(N, 1)
  310 ALCAD(N,2) = XLOAD(N,2)
      DO 320 N = 1,NBCS
      READ(5,8080) NIBND, IBTYP, VALUE
       IF(IBTYP.NE.LABEL1) GOTO 315
       IBOUND(N) = NIBND
       XBOUND(N+1) = VALUE
       GOTO 318
  315 IF(IBTYP.NE.LABEL2) CALL PRER1(9)
       I30UND(N) = -NIBNO
       XBOUND (N.2) = VALUE
  318 ABOUND(N,1) = XBOUND(N,1)
  320 ABOUND (N, 2) = XBOUND (N, 2)
С
С
       DETERMINE BANDWIDTH
С
       MBAND = 0
       NK = 0
       00 510 N = 1, NELH
       D0 500 K = 1.NELV
      NK = NK + 1
       JD = JCNL(JTOP(NK,3),2) - JCNL(JTOP(NK,1),2)
  5CO IF(JD.GT.MBAND) MBAND = JD
```

JD = JCNL(JTOP(NK,4),2) - JCNL(JTOP(NK,2),2)510 IF(JD.GT.MBAND) MBAND = JD  $MBAND = 2 \neq MBAND + 2$ IF(MBAND.GT.MA) GOTO 2020 NUNTDF = 2+JCNL(JTOP(NREL, 3), 2) READ(5, B110) ICHECK WRITE(6.9010) IF( (CHECK .EQ. IEQUI) WRITE( 6,9020) WRITE(6,9030) RETURN END SUBROUTINE OUTPUT C С . THIS SUBROUTINE PRINTS ALL INITIAL INFORMATION (N TABULAR FORM С С £ IMPLICIT REAL+8(A-H.O-Z) COMMON S(10,10), VKH, VKV COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI COMMON NN, NH, MA, NODV, NODH, NUMNOD, NR EL, NT EL, NUMBAR, MAXNDF, MBAND COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO COMMON NUMDE, NUMTOF, NIT, IT, NCUR VB, NZB, NOO2 С COMMON/ELEM / X(650), Y(650), THETA(1152,2), U(1300), TU(1300) COMMON/ELEM1 / JTOP(288,4), THICK(288) COMMON/MODULC/ SEC(10,2), EC(9), XNUC(9), TOL COMMON/MODULS/ SES(20,2),ES(19),XNUS(19) COMMON/MODUL B/ SEB(10,2), EB(9) COMMON/LOADS / XLOAD( 20,21,XBOUND( 20,2) COMMON/LOADS1/ ALOAD( 20,2) ABOUND( 20,2) COMMON/LOADS2/ ILOAD( 20), IBOUND( 20), NLOAD, NBCS COMMON/REINF / SAREA( 50), JCNL(650,2), ITOP( 50,6), NREINF, IS1, IS2 9000 FORMAT(/1X,42(1H\*)/43H \* COORDINATES OF NODES 1/,1X,42(1H\*),/) 9010 FORMAT( NODE NO X-COORDINATE Y-COORDINATE ,/) 9020 FORMAT(1X.15.6X.1PD12.4.5X.1PD12.4) 9025 FORMAT(1X,15,6X,1PD12.4,5X,1PD12.4,5X,\*STEEL\*) 9030 FORMAT(1H1,42(1H+)/43H \* TOPOLOGICAL PROPERTIES OF ELEMENTS \*. 1/,1X,42(1H+),/) 9040 FORMAT( FLEMENT NO NODE A NODE B NODE C NODE D NODE M 1 THICKNESS',/) 9050 FORMAT(1H1.//) 9060 FORMAT(18,4X,15,4(4X,15),5X,1PD12.4) 9070 FORMAT(///. NO REINFORCEMENT IN THIS PROBLEM\*) 9080 FORMAT(1H1,42(1H+)/43H \* ARRANGEMENT OF REINFORCEMENT ۰. 1/, 1X, 42(1H\*), /)9090 FORMAT(" BAR NUMBER NO OF BARS TYPE OF BARS TOTAL AREA 1 FROM JOINT TO JOINT .// 9100 FORMAT (1X, I6, 11X, I3, 11X, 'NO', I3, 6x, 1PU12.4, 5X, I4, 5X, I4) 9110 FORMAT(///,1X, CALCULATED BANDwIDTH: MBAND = ',14) 9120 FORMAT(1H1, 42(1H\*)/43H \* STRESS - STRAIN LAWS \*. 1/,1X,42(1H\*),/) 9130 FORMAT(/, CONCRETE: 15, PGINTS GIVEN //) 9140 FORMAT( POINT SIGMA IN PS1 EPSILON IN IN/INN E-MODUL 1 NU-VALUE BETWEEN POINTS ./ ) 9150 FORMAT(1X, 13, 5X, 1PD12, 5, 8X, 1PD12, 5, /, 50X, 1PD12, 5, 2X, 0PD12, 5, 5X, 12, 1 AND , 12)

9160 FORMAT(1x,13,5x,1PD12.5,8x,1PD12.5,/) 9170 FORMAT(/, \* STEEL: \*, 3X, 15, \* POINTS G(VEN\*, /) 9180 FORMAT(/, BOND: ,4X,15, POINTS GIVEN ,/) 9190 FORMAT(///, NUMBER OF (TERATIONS = ,14,5X, TOLERANCE = ,1PD12.5) 9200 FORMAT(1H1,42(1H+)/43H + LOADING INFORMATION 1/,1X,42(1H\*),/) 9210 FURMAT(1x, LOADS FUR INCREMENT NO! .15./) 92'20 FORMAT (" NODE NO X-LOAD', 10X, 'Y-LOAD IN LBS / J 9230 FORMAT(1H1,42(1H\*)/43H \* BOUNDARY CONDITIONS 1/, 1x, 42(1H\*), /)9240 FORMAT(1x, PRESCRIBED DISPLACEMENTS FOR INCREMENT NO+, 15, /) 9250 FORMAT( NODE NO X-DISPL",7X," Y-DISPL IN INCHES',/) 9255 FORMAT (1x, 15, 8x, 1P 012.5, 4x, 1P012.5) 9260 FORMAT(1X, 15, 8X, 1PD12.5) 9265 FORMAT(1X, 15, 24X, 1PD12.5) 9270 FORMAT(///,1X,"ELASTIC SOLUTION ONLY") 9280 FURMAT(///,1X,"AUTOMATIC SCALING",/,1X,"GIVEN LOADS AND DISPL. ARE 1 ASSUMED AS TOTAL VALUES \*\* 9290 FORMAT(///,1x, 'INDIVIDUAL LOAD AND DISPL. INPUT',/,1X, 'PRUGRAM REA IDS\*, I5. INCREMENTS\*) C С PRINT NODAL INFORMATION С WRITE(6,9000) WRITE(6,9010) KP # 0 IPAGE = 36DO LO NC = 1,NUMNOD IF(KP.NE.IPAGE) GOTO 5 KP = 0IPAGE = 55wRITE(6,9050) WRITE(6.9010) 5 KP = KP + 1WRITE(6,9020) JCNL(NC,2),X(NC),Y(NC) IF(JCNL(NC,1).EQ.0) GOTO 10  $KC = JCNL(NC_{2}) + 1$ WRITE(6,9025) KC.X(NC), Y(NC) KP = KP + 1IF(JCNL(NC,1).EQ.1) GOTO 10 KC = KC + 1WRITE(6,9025) KC,X(NC),Y(NC) KP = KP + 110 CONTINUE С C. PRINT TOPOLOGICAL QUANTITIES С WRITE(6,9030) WRITE(6,9040) KP = 0IPAGE = 45DO 30 IP = 1.NREL IFIKP.NE.IPAGE) GOTO 20 KP = 0IPAGE = 55WR ITE(6,9050) wRITE(6,9040) 20 KP = KP + 130 WRITE(6,9060) IP,(JCNL(JTDP(IP,J),2),J=1,4), JCNL((JTDP(IP.1)+NDDV)

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1,2),THICK(IP)
C
      PRINT REINFORCEMENT INFORMATION
С
c.
      IF(NREINF.NE.0) GOTO 40
      WRITE(6,9070)
      GOTO 60
   40 WRITE(6.9080)
      WRITE(6,9090)
      DG 50 IR = 1,NUMBAR
   50 WRITE(6,9100) IR, ITOP(1R, 1), ITUP(1R, 2), SAREA(IR),
    11TOP(IR, 5) 1TOP(IR,6)
   60 CONTINUE
      WRITE(6,9110) MBAND
С
      PRINT CONSTITUTIVE LAWS FOR CONCRETE AND STEEL
С
C.
      WRITE(6,9120)
      WRITE(6,9130) NCURVC
      WRITE(6,9140)
      NCURV = NCURVC - 1
      DO 70 NC = 1, NCURV
      NE = NC + 1
   70 WRITE(6,9150) NC, SEC(NC,1), SEC(NC,2), EC(NC), XNUC(NC), NC, NE
      WRITE(6,9160) NE,SEC(NE,1),SEC(NE,2)
      IF(NREINF.EQ.0) GOTO 90
      WRITE(6,9120)
      WRITE(6,9170) NOURVS
      WRITE(6,9140)
      NCURV = NCURVS - 1
      DO 80 NS = 1,NCURV
      NE = NS + 1
   80 WRITE(6,9150) NS, SES(NS, 1), SES(NS, 2), ES(NS), XNUS(NS), NS, NE
      WRITE(6,9160) NE,SES(NE,1),SES(NE,2)
      WRITE(6,9120)
      WRITE(6,9180) NCURVB
      WRITE(6,9140)
      NCURV = NCURVB - 1
      DO 85 NB = 1,NCURV
      NE = NB + 1
   85 WRITE(6,9145) NB, SE8(NB,1), SEB(NB,2), EB(N8), NB, NE
      WRITE(6,9160) NE, SEB(NE, 1), SEB(NE, 2)
9145 FURMAT(1x,13,5x,1P012.5,8x,1P012.5,/,48x,*BOND STIFFNESS *,1P012.5
     1,5X,12, * AND *,12)
      IF(VKV.EQ.0.0) GOTO 86
      WRITE(6,9155) VKV
 9155 FORMAT(//,1x, *SPECIFIED DOWEL ACTION STIFFNESS KV = *,1PD12.5)
      GOTO 90
   86 MRITE(6,9165)
9165 FORMAT(//,1X, DOWEL ACTION STIFFNESS CHOSEN AS (10**10)*S(1,1)')
c
      PRINT ITERATION AND TOLERANCE VALUES
С
c
   90 WRITE(6.9190) NIT, TOL
      IF(NINCR - 1) 91,92,93
   91 WRITE(6,9270)
      GOTO 95
   92 WRITE(6,9280)
      GOTO 95
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93 WRITE(6.9290) NINCR
    PRINT LOADING INFURMATION
 95 WRITE(6,9200)
   WRITE(6,9210) KINCR
   WRITE(6.9220)
   DO 100 IL = 1, NLOAD
100 WRITE(6.9255) JCNL(ILOAD(IL).2) , XLOAD(IL.1) , XLOAD(IL.2)
   PRINT BOUNDARY CONDITIONS
   WRI TE(6,923D)
   WRITE(6,9240) KINCR
   WRITE(6,9250)
   DO 110 IB = 1,NBCS
   NIBND = IBOUND(IB)
   IF(NIBND.LE.0) GOTO 105
   WRITE(6,9260) JCNL(N1BND,2) , XBOUND(IB,1)
   GOTO 110
105 NIBND = -NIBND
    WRITE(6,9265) JCNL(NIBND,2) , XBOUND(18,2)
110 CUNTINUE
   RETURN
   END
   SUBROUT INE STIFF(ISW1CH,MTYP,J,I,JJ1,JJ2,JJ3,K1,K2,K3)
 .
 . THIS SUBROUTINE CALCULATES ALL ISOTROPIC AND ANISOTROPIC
 . STIFFNESS MATRICES FOR THE CONCRETE PANELS, STEEL BARS AND
 . AND BOND LINKS.
      PARAMETERS AND ARRAYS:
 • S(8,8) = STIFFNESS MATRIX
 SS(8,8) = AUXILIARY STIFFNESS MATRIX
 ST(8,8) = AUXILIARY STIFFNESS MATRIX
 . ROT(8,8) = ROTATIONAL TRANSFORMATION MATRIX
  IMPLICIT REAL*8(A-H,0-Z)
   REAL *8 NUA, NUB, NUAB, NUAB2, DSQRT, DCUS, DSIN
   COMMON S(10,10),VKH,VKV
   COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI
   COMMON NN, NH, MA, NODV, NODH, NUMNOD, NREL, NTEL, NUMBAR, MAXNOF, MBAND
   COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO
   COMMON NUMDE, NUMTOF, NIT, IT, NCURVE, NZB, NOD2
   COMMON/ELEM / X(650),Y(650),THETA(1152,2),U(1300),TU(1300)
   COMMON/ELEM1 / JTOP(288,4), THICK(288)
   COMMON/TYP / ITYPE(1152), ISTYPE(50), IBTYPE(100), KDIR
   COMMON/MODULC/ SEC(10,2),EC(9),XNUC(9),TOL
   COMMON/MODULS/ SES(20,2),ES(19),XNUS(19)
   COMMON/MODULB/ SEB(10,21,EB(9)
   COMMON/REINF / SAREA( 50), JCNL(650,2), ITOP( 50,6), NREINF, IS1, IS2
   DIMENSION ROT (10,10) , SS(10,10) , ST(10,10)
   DIMENSION H(10,3), BB(3,10), DD(3,3)
   DATA ZER0/0.0D+00/ , R0T/100+0.0D+00/ , D180/180.0D+00/
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C

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C

C.

C.

C.

С

9 DD(L,M) = ZERO 00.8 M = 1,108 88(L,M) = ZERO IFININCR.EQ.11 GOTO 19 CALL TYPE(J,MTYP) 19 CONTINUE GOTO (10,200,230) , MTYP 10 IF( ISWICH.NE.1) GOTO 30  $D0 \ 20 \ L = 1,10$ DO 20 M = 1.10 20 S(L,M) = 0.0 3C ITYP = ITYPE(J)IF(ITYP.GT.1) GOTO 35 X32 = X(JJ3) - X(JJ2)= X(JJ3) - X(JJ1) X21 = X(JJ2) - X(JJ1) $Y_{1} = Y_{1} = Y_{1} = Y_{1}$ Y31 = Y(JJ3) - Y(JJ1)Y21 = Y(JJ2) - Y(JJ1)GOTO 38 35 DMEGA = THETA(J.1)\*PI/0180 OMEGAL = DCOS(OMEGA) DMEGA2 = DSIN(OMEGA) CHI1 = OMEGA1\*X(JJ1) + OMEGA2\*Y(JJ1) ETA1 = -OMEGA2\*X(JJ1) + OMEGA1\*Y(JJ1) CHI2 = OMEGA1\*X(JJ2) + OMEGA2\*Y(JJ2) FTA2 = -OMEGA2 \* X(JJ2) + OMEGA1 \* Y(JJ2)CHI3 = OMEGA1\*X(JJ3) + OMEGA2\*Y(JJ3) ETA3 = -OMEGA2\*X(JJ3) + OMEGA1\*Y(JJ3) X32 = CHI3 - CHI2Y32 = EIA3 - EIA2X31 = CHI3 - CHI1 Y31 = ETA3 - ETA1 x21 = CHI2 - CHI1¥21 = ETA2 - ETA1 38 KJ2 =  $2 \times K1$ KJ1 = KJ2 - 1KJ4 = 2 \* K2KJ3 = KJ4 - 1KJ6 = 2 \* K3KJ5 = KJ6 - 1A1 = 0.5\*(X32\*Y21 - X21\*Y32)AR = 1.0/(2.0\*A1) 88(1,KJ1) = AR\*Y32 BB(1,KJ3) = -AR\*Y31 BB(1,KJ5) = AR + Y21 BB(2,KJ2) = -AR + X32BB(2+KJ4) = AR\*X31 BB(2,KJ6) = -AR + X21BB(3,KJ1) = BB(2,KJ2)BB(3,KJ2) = BB(1,KJ1)BB(3,KJ3) = BB(2,KJ4) BB(3,KJ4) = BB(1,KJ3)BB(3,KJ5) = BB(2,KJ6)BB(3,KJ6) = BB(1,KJ5)

 $00 \ 8 \ L = 1,3$ 

D0.9 M = 1.3

IF(ITYP.GT.1) GOTO 100

C

C ċ ELASTIC STIFFNESS Ċ. CN = E1 \* THICK(I)/(1.0 - CNU \* CNU) $DD(1,1) = CN \neq A1$ DD(1,2) = CN\*CNU\*A1 DD(2,1) = DD(1,2)DD(2,2) = DD(1,1)DD(3,3) = CN+0.5+(1.0 - CNU)+A1 C BBT\*DD\*88 С c 50 DD 2 L = 1.10 DO 2 M = 1,3H(L,M) = 0.0DO 2 K = 1,3 2 H(L, H) = H(L, H) + BB(K, L)\*DO(K, H) IF(ITYP.GT.1) GOTO 4 DO 3 L = 1,10 D3 3 M = 1,10DO 3 K = 1,3 3 S(L,H) = S(L,H) + H(L,K)\*BB(K,H) RETURN 4 DD 5 L = 1.10D0.5 M = 1,10SS(L,M) = ZEROD0.5 K = 1.35 SS(L,M) = SS(L,M) + H(L,K) +BB(K,M) C ROTATE ANISOTROPIC OR CRACKED STIFFNESS MATRIX C Ċ DO 150 L = 1, 10, 2ROT(L,L) = OMEGA1  $ROT(L_{+}L+1) = OMEGA2$ ROT(L+1,L) =-OMEGAZ 150 ROT(L+1,L+1) = OMEGA1 DO 170 L = 1.10DO 170 M = 1,10 ST(L,M) = 0.0DO 160 LM = 1,10160 ST(L,M) = ST(L,M) + ROT(LM,L)\*SS(LM,M) 170 CONTINUE 190 L = 1,10DO 190 M = 1,10STEMP = 0.0DO 180 LM = 1,10 180 STEMP = STEMP + ST(L,LM)\*RUT(LM,M) 190 S(L,M) = S(L,M) + STEMP . RETURN C. CRACKED ELEMENT c C 100 CONTINUE C ANISOTROPIC OR CRACKED ELEMENT STIFFNESS MATRIX С C CA = 1.0/(1.0 - CNU12\*CNU21)\*THICK(I) DD(1,1) = E1 \* CA

DD(1.2) = E1\*CA\*CNU12

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DD(2,1) = DD(1,2)
      DD(2,2) = E2+CA
      DD(3,3)= DSQRT((E1/(1.0+CNU12))*(E2/(1.0+CNU21))/4.0)*THICK(I)
      IF(ITYPE(J), EQ.4) DD(3,3) = ZERO
      60 10 50
С
C
      THIS PART OF STIFF CALCULATES THE MEMBER STIFFNESS MATRIX
      FOR THE REINFORCEMENT BARS
С
  200 XX = X(JJ2) - X(JJ1)
      YY = Y(JJ2) - Y(JJ1)
      XLS =DSQRT(XX*XX + YY*YY)
      CX = XX/XLS
      CY = YY/XLS
      C = E1*SAREA(J)/XLS
      CX2 = CX \neq CX
      CY2 = CY + CY
      CXY = CX + CY
C
      S(1,1) = C+CX2
      S(1,2) = C + C X Y
      S(1,3) = -S(1,1)
      S(1,4) = -S(1,2)
      S(2,2) = C+CY2
      S(2,3) = S(1,4)
      S(2,4) = -S(2,2)
      S(3,3) = S(1,1)
      S(3,4) = S(1,2)
      S(4,4) = S(2,2)
  210 DO 220 IR = 1,4
      DO 220 JR = 1.4
  220 S(JR, IR) = S(IR, JR)
      RETURN
C
      STIFFNESS OF BOND LINKS
C
      ISWICH = 1 HORIZONTAL
С
      ISWICH = 2 VERTICAL
£.
С
  230 DO 240 IB = 1,4
      DO 240 JB = 1,4
  240 S(IB, JB) = 0.0
      IF(ISWICH.EQ.1) GOTO 250
C
      VERTICAL STEEL BAR
      C0 = 0.0
      SI = 1.0
      GOTO 260
      HORIZONTAL STEEL BAR
С
  250 CO = 1.0
      SI. = 0.0
  260 C2 = C0 * C0
      S2 = SI \neq SI
      SC = SI + CO
      S(1,1) = VKH \times C2 + VKV \times S2
      S(1,2) = VKH*SC - VKV*SC
      S(1,3) = -S(1,1)
      S(1,4) = -S(1,2)
      S(2,2) = VKH+S2 + VKV+C2
      S(2,3) = S(1,4)
      S(2.4) = -S(2.2)
```

```
S(3,3) = S(1,1)
      S(3,4) = S(1,2)
      S(4,4) = S(2,2)
      GOTO 210
С
      END
      SUBROUTINE TYPE(J,MTYP)
C
С
    c
    . THIS SUBROUTINE CALCULATES THE PROPER MODULI OF ELASTICITY
C
    . AND ASSIGNS, THE PARAMETER ITYPE TO THE ELEMENTS
c
    • ITYPE = 1 ISOTROPIC, ELASTIC
• ITYPE = 2 ANISUTROPIC, ELASTIC
С
C.
ſ.
    ITYPE = 3 ANISOTROPIC, PLASTIC-
ſ.
C
    С
C
      IMPLICIT REAL*8(A-H,G-Z)
      COMMON S(10,10), VKH, VKV
      COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI
      COMMON NN, NH, MA, NODV, NODH, NUMNOD, NREL, NTEL, NUMBAR, MAXNDF, MBAND
      COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO
      COMMON NUMDE, NUMTOF, NIT, IT, NCURVE, NZE, NOD2
      COMMON/TYP / ITYPE(1152), ISTYPE(50), IBTYPE(100), KDIR
     COMMON/MODULC/ SEC(10,2),EC(9),XNUC(9),TOL
COMMON/MODULS/ SES(20,2),ES(19),XNUS(19)
      COMMON/MODUL 8/ SEB(10,2),EB(9)
     COMMON/CONSTR/ STR(1152,3), STRP(1152,2), EPS(1152,3), EPR(1152,2)
     COMMON/STLSTR/ ESPS( 50), STRS( 50), TESPS( 50), TSTRS( 50,2)
      COMMON/BOND / EPSB(100,2), STKB(100,2), TEPSB(100,2), TSTRB(100,2)
     DATA D10/1.00-10/ , D20/1.0D-20/ , ZER0/0.0D+00/
 9000 FORMAT (//,1X, *CONCRETE ELEMENT NO.*,16)
 9010 FORMAT(//,1X, STEEL ELEMENT NO. +,16)
С
С
      NINCR'= 1 CALLS FOR AUTOMATIC SCALING
      TYPE MUST REMAIN UNCHANGED (ITYPE(J) = 1) FOR PROPER SCALING
C
C
      KPRER = 0
      IF(MTYP - 2) 10,120,210
С
      DETERMINE TYPE OF CONCRETE ELEMENTS
С
   10 K = 1
      II = NZC
      I2 = NZC
      EP1 = EPR(J_1)
      EP2 = EPR(J_{1}2)
     GO TO 30
   20 K ≠ 2
      EP1 = EP2
      I1 = I2
   30 IF ( EP1 .GE. 0.0000 ) GU TU 70
     I2 = NZC - 1
   50 IF ( EP1 .GE. SEC(12,2) ) GO TO 60
     I2 = I2 - 1
      IF(12.GE.1) GOTO 50
```

```
KPRER = 1
      I2 = 1
   60 IF(K-1) 65,20,100
   65 WRITE(6,2000) J, MTYP
 2000 FORMAT & / * STOP AT J =*, I5, * MTYP =*, I3 }
      STOP
   70 12 = N2C
      IF(EP1.LT.SEC(12+1,2)) GOTO 90
С
C
      CRACKED ELEMENT
C
      ITYPE(J) = 4
      E1 = ZERO
      CNU12 = ZERO
      I2 = NZC
      CNU21 = ZERO
      IF(EP2.LT.0.0000) GOTO 75
      IF(EP2.LT.SEC(NCURVC,2)) GOTO 78
      WRITE(6,7310)
 7310 FORMAT( DOUBLY CRACKED ELEMENT, PROGRAM STOPS )
      CALL EXIT
С
      EP 2 NEGATIVE
   75 12 = NZC - 1
   76 IF(EP2.GE.SEC(12,2)) GOTO 78
      12 = 12 - 1
      IF(12.GE.1) GOTO 76
      KPRER ≠ 1
      12 = 1
   78 E2 = EC(12)
      GOTO 110
   90 IF(K.EQ.1) GOTO 20
  100 E2 = EC(12)
     E1 = EC(11)
      IF(E1.LT.E2) GOTO 101
С
      ASSIGN NU12
      CNU12 = XNUC(11)
      CNU21 = E2/E1 + CNU12
      GOT 0 106
C
      ASSIGN NU21
  101 CNU21=XNUC(12)
      CNU12 = E1/E2 \neq CNU21
  106 \text{ ITYPE(J)} = 3
      IF((11.LT.(NZC-1)).OR.(11.GE.(NZC+1))) GOTO 110
      IF((12.LT.(NZC-1)).DR.(12.GE.(NZC+1))) GOTO 110
      ITYPE(J) = 1
      IF((11.EQ.N2C).AND.(12.EQ.11.0R.12.EQ.N2C-1)) GO TO 110
      IF((I1.EQ.NZC-1).AND.(12.EQ.I1.DR.12.EQ.NZC)) GO TO 110
      ITYPE(J) = 2
  110 CONTINUE
      CNU
                = CNU12
      IF(KPRER.EQ.0) RETURN
      WRITE(6.9000) J
      CALL PRERZ(KPRER)
      RETURN
.C
      DETERMINE TYPE OF STEEL ELEMENTS
C.
C
 120 II = NZS
      EP1 = TESPS(J)
```

```
1F ( EP1 .GE. 0.0000 ) GO TU 160
      I1 = NZS - 1
  140 IF ( EP1 .GE. SES( 11, 2) ) GO TO 190
      11 = 11 - 1
      IF(I1.GE.1) GOTO 140
      KPRER = 1
      11 = 1
      GO TO 190
  160 I1 = N2S
  170 IF ( EP1 .LE. SES(11+1,2) ) GO TO 190
      \Pi = \Pi + 1
      IF(I1.LE. (NCURVS-1)) GOTO 170
      KPRER = 2
      II = NCURVS - 1
  190 E1 = ES(I1)
      SNU = XNUS(I1)
      ISTYPE(J) = 2
      IF ( { II .EQ. NZS } .OR. ( II .EQ. ( NZS-1) ) | ISTYPE(J) = 1
      IF ( KPRER .EQ. 0 ) RETURN
      WRITE(6,9010) J
      CALL PRER2(KPRER)
      RETURN
С
      DETERMINE TYPE OF BOND ELEMENTS
C
C
  210 II = NZB
      IF ( EP1 .GE. 0.0000 ) GO TO 250
      I1 = NZB - 1
  230 IF ( EP1 .GE. SE8(11,2) ) GO TO 280
      11 = 11 - 1
      IF(I1 - 1) 270,230,230
  250 I1 = NZB
  260 IF ( EP1 .LE. SEB(11+1,2) ) GO TO 280
      11 = 11 + 1
      IF(11.LE. (NCURV8-1)) GOTO 260
С
      BOND LINK BREAKES DOWN, ASSIGN ZERO STIFFNESS
С
C
      IN DIRECTION OF FAILURE
c
  270 VKH = ZERU
      18TYPE(J) = 3
      RETURN
C
      ACTING BOND LINK VKH = STIFFNESS PARALLEL TO BAR
С
                       VKV = STIFFNESS PERPENDICULAR TO BAR.
C
                             ASSIGNED FOR ALL LINKS IN MAIN
С
C.
  280 \text{ VKH} = \text{EB(I1)}
      IBTYPE(J) = 2
      IF((I1.EQ.(NZB-1)).OR.(I1.EQ.NZB)) IBTYPE(J) = 1
      RETURN
      END
      SUBROUTINE BANSOL
С
      SUBROUTINE BANSOL SOLVES BANDED MATRICES BY THE GAUSS ELIMINATION
С
      PROCEDURE. THE BANDMATRIX IS STORED DIRECTLY IN A RECTANGULAR
С
С
      ARRAY AND TREATED BLOCKWISE.
c
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IMPLICIT REAL#8(A-H,C-Z)
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COMMON S(10,10),VKH,VKV CONMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, P1 COMMON NN, NH, MA, NODV, NUDH, NUMNOD, NREL, NTEL, NUMBAR, MAXNDF, MBAND CONMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO COMMON NUMDE, NUMTOF, NIT, IT, NCURVE, NZB, NOD2 COMMON/ POOL / A(108, 54), B(108), NUMBLK NB = 0 NL = NN + 1REWIND 1 REWIND 2 GOTO 2 SHIFT LOWER PART OF B AND A INTO UPPER PART CLEAR LOWER PART 1 NB = NB + 1 DO 3 N = 1.NN NM = NN + NB(N) = B(NM) B(NM) = 0.0DO 3 M = 1.MBANDA(N,M) = A(NM,M) 3 A(NM+M) = 0.0 IF(NUMBLK - N8) 2.4.2 READ LOADS AND STIFFNESS INTO LOWER PART OF A AND B 2 READ(2) ( B(N), (A(N, M), M=1, MBAND), N=NL, NH) IF(NB) 4,1,4 REDUCE UPPER PART 4 CONTINUE DO 5 N = 1.NN IF (A(N,1)) 6,5,6 6 B(N) = B(N)/A(N-1)DO 7 L = 2, MBAND IF(A(N.L)) 8.7.8 8 Q = A(N,L)/A(N,1) I = N + L - LJ = 0 DO 9 K = L.MBAND J = J + 1 $9 A(I_J) = A(I_J) - Q A(N_K)$ B(I) = B(I) - A(N,L)+B(N) A(N,L) = Q7 CONTINUE 5 CONTINUE IF(NUMBLK - NB) 10.11.10 WRITE LOADS AND STIFFNESS ON TAPE NUMBER 1 10 WRITE(1) (B(N), (A(N,H), M=2, MBAND), N=1, NN) GOTO 1 BACKSUBSTITUTION STARTS WITH LAST BLOCK STILL IN CURE

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11 CONTINUE 00 12 M = 1.NN N = NN + 1 - M00 13 K = 2. MBAND L = N + K - 113 B(N) = B(N) - A(N.K)+B(L) NM = N + NNB(NM) = B(N)12 A(NM, NB) = B(N) NB = NB - 1IF(NB) 14,15,14 BACKSPACE TAPE ONE RECORD AND READ NEW BLOCK 14 BACKSPACE 1 READ(1) (B(N), (A(N, M), M=2, MBAND), N=1, NN) BACK SPACE 1 GOTO 11 15 RETURN END SUBROUTINE MODIFY(I, USTAR) THIS SUBROUTINE MODIFIES THE MAIN STIFFNESS MATRIX FOR PRESCRIBED DISPLACEMENTS U(I) = USTAR THE ELEMENTS OF THE I-TH ROW AND COLUMN ARE SET EQUAL TO ZERO THE ELEMENT IN THE DIAGONAL IS SET EQUAL TO 1.0 = POSITION OF DISPLACEMENT RELATIVE TO BLOCKBEG PARAMETERS: I USTAR . VALUE OF THE DISPLACEMENT IMPLICIT REAL+8(A-H,O-Z) COMMON S(10,10), VKH, VKV COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI COMMON NN, NH, MA, NODY, NODH, NUMNOD, NREL, NTEL, NUMBAR, MAX NDF, MBAND COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO COMMON NUMDE, NUMTOF, NIT, IT, NCURVB, NZB, NOD2 COMMON/POOL / A(108,54),B(108);NUMBLK DO 30 J = 1, MBAND  $\mathbf{K} = \mathbf{I} - \mathbf{J} + \mathbf{I}$ IF(K.LE.0) GOTO 20 B(K) = B(K) - A(K, J)+USTAR A(K,J) = 0.020 K = I + J - 1IF(K.GT.NH) GOTO 30 B(K) = B(K) - A(I,J)+USTAR  $A(I_J) = 0.0$ 30 CONTINUE  $A(I_{1}I) = 1.0$ B(I) = USTAR RETURN END SUBROUTINE SCALE SUBROUTINE SCALE ADJUSTS THE LOAD VECTOR ALOAD AND THE PRESCRIBED DISPLACEMENT VECTOR AFTER THE FIRST SOLUTION STEP FOR NINCR = 1 THE LOAD INCREMENTS ARE AUTOMATICALLY CALCULATED AND PLACED IN XLOAD, ALUAD CONTAINS THE CURRENT LOAD APPLIED IF THE STRAINS ARE LESS THAN THE ELASTIC LIMIT, THE PROGRAM

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STOP'S AFTER THE FIRST RUN FOR NINCE GT 1 THE PROGRAM READS THE INDIVIDUAL LOAD INCREMENTS AND ADDS THEM TO ALOAD IMPLICIT REAL+8(A-H, 0-2) CDMMON \$ (10.10) . VK H. VKV COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI COMMON NN, NH, MA, NODV, NODH, NUMNOD, NREL, NTEL, NUMBAR, MAXNDF, MBAND CUMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO COMMON NUMDE, NUMTOF, NIT, IT, NCURVB, NZB, NOD2 COMMON/MODULC/ SEC(10,2).EC(9).XNUC(9).TOL COMMON/MODULS/ SES (20,2), ES (19), XNUS (19) COMMON/MODULB/ SEB(10,2),EB(9) COMMON/LOADS / XLOADI 20,21, XBOUNDI 20,21 COMMON/LOADS1/ ALOAD( 20,2), ABOUND( 20,2) COMMON/LOADS2/ ILOAD( 20), IBOUND( 20), NLOAD, NBCS COMMON/REINF / SAR EA1 50), JCNL(650, 2), ITOP( 50,6), NREINF, IS1, IS2 COMMGN/CONSTR/ STR(1152,3), STRP(1152,2), EPS(1152,3), EPR(1152,2) COMMON/CONST1/ TSTR(1152,3), TEPS(1152,3) COMMON/STLSTR/ ESPS( 50).STRS( 50).TESPS( 50).TSTRS( 50.2) SEARCH FOR LARGEST STRAIN AT THE END OF THE FIRST SOLUTION ECMAX = 0.0 ETMAX = 0.0 DO 1 KSCALE = 1.NTEL IF(STRP(KSCALE,1) - ETMAX) 2,2,20 20 ETMAX = STRP{KSCALE.1] IELNAX = KSCALE 2 IF(STRP(KSCALE,2) - ECMAX) 100,1,1 100 ECMAX = STRP(KSCALE,2) JELMAX = KSCALE 1 CUNTINUE FIND THE PROPER SCALING FACTORS XSCAL1 = SEC(NZC+1.1)/ETMAX XSCAL2 = SEC(NZC-1,1) /ECMAX WRITE(6,2000) IELMAX,STRP(IELMAX,1),XSCAL1,JELMAX,STRP(JELMAX,2), 1 XSCAL2 2000 FORMAT(///, SCALING INFORMATION,//, ELEMENT NO, 15, PRINCIPAL 1 STRESS: SP1 = ",1P012.5," SCALE FACTOR = ",1P010.3,/," ELEMENT 2 NO +15+ PRINCIPAL STRESS: SP2 = ++1PD12.5+ SCALE FACTOR = +. 31PD10.3) IF((XSCAL1.GT.1.0).OR.(XSCAL2.GT.1.0)) GOTO 3 LOAD YIELDS STRAINS LARGER THAN ELASTIC LIMIT XLOAD CONTAINS INCREMENTAL LOADS ALDAD CONTAINS LOAD AT LIMIT AND SUBSEQUENT TOTAL LOADS XSCAL = XSCALZ ¥ NINCR = 2\*XSCAL IF(XSCAL1.LT.XSCAL2) XSCAL = XSCAL1 DECREASE LOADS, CALCULATE INCREMENTS DO 5 IL = 1, NLOADALOAD(IL,1) = XLOAD(IL,1)\*XSCAL ALOAD(IL,2) = XLOAD(IL,2)\*XSCAL XLOAD(IL,1) = (XLOAD(IL,1) - ALOAD(IL,1))/NINCR 5 XLOAD(IL,2) = (XLOAD(IL,2) - ALOAD(IL,2))/NINCR DO 6 IB = 1.NBCS ABOUND(IB,1) = XBOUND(IB,1)\*XSCAL

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ABOUND(IB,2) = XBOUND(IB,2)\*XSCAL XBOUND(IB,1) = (XBOUND(IB,1) - ABOUND(IB,1))/NINCR 6 XBQUND(IB,2) = (XBOUND(IB,2) - ABOUND(IB,2))/NINCR IAUTO = 1· · . GOTO 14 LARGEST STRAIN IS LESS THAN ELASTIC LIMIT c. 3 IF(NINCR - 1) 8,9,10 HRONG NINCR, CHANGED TO 1 8 NINCR = 1KPRER2 = 3CALL PRER2 (KPRER2) TOTAL LOAD APPLIED DOES NOT GIVE INELASTIC BEHAVIOR C. ELASTIC SOLUTION ONLY 9 KPRER2 = 4 CALL PRER2(KPRER2) 11 CONTINUE STOP C FIRST INCREMENT IS TOO SMALL TO YIELD INELASTIC STRAINS c READ NEXT INCREMENT r 10 CONTINUE С ċ ENTRY SCALE2 IF(KINCR.EQ.NINCR) GOTO 11 00 12 IL = 1, NLOAD IF(IAUTO.EQ.1) GOTO 122 READ(5,1000) KLOAD, XLOAD(IL,1), XLOAD(IL,2) IF(KLOAD.EQ.ILOAD(IL)) GOTO 122 WRITE(6,1100) IL 11CO FORMAT( + LOAD ERROR AT LOAD + 15) CALL EXIT 1000 FORMAT(15.2F12.4) 122 ALOAD(IL, 1) = ALOAD(IL, 1) + XLOAD(IL, 1) 12 ALDAD(IL,2) = ALDAD(IL,2) + XLDAD(IL,2) READ(5.1919) KBCS 1919 FURMAT(15) IF(KBCS.EQ.0) GOTO 14 DO 13 IB = 1,NBCS IF( IAUTO.EQ.1) GOTO 133 READ (5,1000) IBOUND (IB), XBOUND (IB, 1), XBOUND (IB, 2) 133 ABOUND(IB,1) = ABOUND(IB,1) + XBOUND(IB,1) 13 ABOUND(IB,2) = ABOUND(IB,2) + XBOUND(IB,2) 14 CONTINUÉ KINCR = KINCR + 1 WRITE(6,5020) 5020 FORMAT(1H1,42(1H\*)/43H \* LOADING INFORMATION 1/.1X.42(1H\*)./) WRITE(6,5220) NINCR,KINCR 5220 FORMAT(1X, TOTAL NUMBER OF INCREMENTS', 15///1X, LOADS FOR INCREMEN 1T NO.", 15,/) WRITE (6,5021) 5021 FORMAT ( NODE NG X-LUAD ,10X, Y-LUAD IN LBS\* / J DO 62 IL = 1, NLOAD62 WRITE(6,5032) JCNL(ILOAD(IL),2),XLOAD(IL,1),XLOAD(IL,2) WRITE(6,9010) KINCR 9010 FORMAT(///,1x, 'TOTAL LOADS FOR INCREMENT', 15,/) WRITE(6,5021)

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DO 602 IL # 1,NLGAD
602 WRITE(6,5032) JCNL(ILUAD(IL),2),ALOAD(IL,1),ALOAD(IL,2)
    IF(KBCS.EQ.0) RETURN
    WRITE(6,5030)
5030 FORMAT(1H1, 42(1H*)/43H * BOUNDARY CONDITIONS
                                                                    *,
   1/,1X,42(1H*),/)
    WRITE(6,5331) NINCR, KINCR
5331 FORMAT(1X. TOTAL NUMBER OF INCREMENTS', 15.///.1X. DISP. FOR INCREM
   1 ENT
           NO.*,15,/)
    WRITE(6,5031)
5031 FORMAT(* NODE NO
                           X-DISPL 7X. Y-DISPL
                                                    IN INCHES ./)
    DO 61 IB = 1.NBCS
  61 WRITE(6,5032) IBOUND(IB), XBOUND(IB,1), XBOUND(IB,2)
5032 FORMAT (1X, 15,8X, 1PD12.5,4X, 1PD12.5)
     WRITE(6,9020) KINCR
9020 FORMAT(///.1X. TOTAL DISPL. FOR INCREMENT'.15./)
    WRITE(6,5031)
    DO 612 IB = 1,N8CS
612 WRITE(6,5032) IBOUND(IB), ABOUND(IB,1), ABOUND(IB,2)
    RETURN
    END
    SUBROUTINE STRESS(1, J, JJ1, JJ2, JJ3, MTYP)
  = NUMBER OF RECTANGULAR ELEMENT PROCESSED
  • I
  • J
           = NUMBER OF TRIANGULAR ELEMENT PROCESSED
  • JJ1
           - NUMBER OF NODE 1
  . JJ2
           = NUMBER OF NODE 2
  • JJ3
           = NUMBER OF NODE 3
  • MTYP = MATERIAL TYPE = 1 FOR CONCRETE
                             * 2 FOR STEEL
  IMPLICIT REAL+8(A-H, D-Z)
    COMMON S(10,10), VKH, VKV
    COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI
    COMMON NN, NH, MA, NODY, NODH, NUMNOO, NR EL, NT EL, NUMBAR, MAXNOF, MBAND
    COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO
    COMMON NUMDE, NUMT DE, NIT, IT, NCUR VS, NZB, NOD2
    COMMON/ELEM / X(650), Y(650), THETA(1152,2), U(1300), TU(1300)
    COMMON/ELEM1 / JTOP(288,4), THICK(288)
    COMMON/TYP / ITYPE(1152), ISTYPE(50), ISTYPE(100), KDIR
    COMMON/MODULC/ SEC{10,2}.EC(9),XNUC(9),TOL
COMMON/MODULS/ SES(20,2),ES(19),XNUS(19)
    COMMON/MODULB/ SEB(10,2),EB(9)
    COMMON/REINF / SAREA( 50), JCNL(650, 2), ITOP( 50,6), NREINF, IS1, IS2
    COMMON/CONSTR/ STR(1152,3), STRP(1152,2), EPS(1152,3), EPR(1152,2)
    COMMON/CONST1/ TSTR(1152,3), TEPS(1152,3)
    COMMON/STLSTR/ ESPS( 50), STRS( 50), TESPS( 50), TSTRS( 50,2)
    COMMON/BOND / EPSB(100,2), STRB(100,2), TEPSB(100,2), TSTRB(100,2)
    DATA D20/1.00-20/ , ZER0/0.0D 00/ , D10/1.0D-10/
    DIMENSION H(3,6) , UU(6) , D(3,3) , JC(3)
    CALL TYPE(J,MTYP)
    IF(MTYP.NE.1) GDTO 110
    DO 10 LH = 1,3
    DO 10 MH = 1,6
  10 H(LH, MH) = 0.0
    00 20 LD = 1.3
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20 D(LD,MD) = 0.0
   ALPHA = 0.0
   BETA = 0.0
   JC(1) = JJ1
   JC(2) = JJ2
   JC(3) = JJ3
   CB = 1.0/((X(JJ3)-X(JJ2))*(Y(JJ2)-Y(JJ1)) -
             (X(JJ2)-X(JJ1))*(Y(JJ3)-Y(JJ2)))
  1
   SIGMA = D+H+U
   DO 25 K = 1.3
   JI = 2 + JCNL(JC(K), 2)
   UU(2*K - 1) = U(JI - 1)
25 UU(2*K) = U(JI)
   SET UP H - MATRIX
   H(1,1) = CB \neq (Y(JJ3) - Y(JJ2))
   H(1,3) = -CB \neq (Y(JJ3) - Y(JJ1))
   H(1,5) = CB*(Y(JJ2) - Y(JJ1))
   H(2,2) = -CB + (X(JJ3) - X(JJ2))
   H(2,4) = CB \neq (X(JJ3) - X(JJ1))
   H(2+6) = -CB*(X(JJ2) - X(JJ1))
   H(3,1) = H(2,2)
   H(3_{2}) = H(1_{1})
   H(3,3) = H(2,4)
   H(3,4) = H(1,3)
   H(3,5) = H(2,6)
   H(3,6) = H(1,5)
   00 40 L = 1.3
   ETEMP = 0.0
   DO 30 M = 1.6
30 ETEMP = ETEMP + H(L,M)+UU(M)
40 \text{ EPS(J,L)} = \text{ETEMP}
   CALCULATE PRINCIPAL STRAINS
   EPSX = TEPS(J,1) + EPS(J,1)
   EPSY = TEPS(J,2) + EPS(J,2)
   EPSXY = TEPS(J,3) + EPS(J,3)
   RAD = ((EPSX - EPSY)/2.0)**2 + (0.5*EPSXY)**2
   EMAX = DSQRT(RAD)
   EAVR = (EPSX + EPSY)/2.0
   EPR(J_1) = EAVR + EMAX
   EPR(J.2) = EAVR - EMAX
   IF(DABS(EPSX - EPSY).LT.D20) GOTO 50
   TAN2A = EPSXY/(EPSX - EPSY)
   ALPHA = DATAN(TAN2A) +90.0/PI
   THETA(J,1) = ALPHA
   UPDATE TYPE
50 CALL TYPE(J,MTYP)
   IF(ITYPE(J).NE.1) GOTO 80
   ELASTIC , ISOTROPIC
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DO 20 MD = 1,3

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CD = E1/(1.0 - CNU+CNU)
      D(1,1) = CD
      D(1,2) = CD+CNU
      D(2,1) = D(1,2)
      D(2,2) = CD
      D(3,3) = CD = 0.5 = (1.0 - CNU)
С
       DO 70 L = 1,3
      STEMP = 0.0
       00 60 M = 1.3
   60 STEMP = STEMP + D(L,M)*EPS(J,M)
   70 STR(J,L) = STEMP
C
      PRINCIPAL STRESSES
С
С
      SIGX = TSTR(J,1) + STR(J,1)
      SIGY = TSTR(J,2) + STR(J,2)
      SIGXY = TSTR(J,3) + STR(J,3)
      RAD = ({SIGX - SIGY}/2.0) + 2 + SIGXY + 2
      SMAX = DSQRT(RAD)
       SAVR = (SIGX + SIGY)/2.0
      STRP(J,1) = SAVR + SMAX
      STRP(J.2) = SAVR - SMAX
      IF(DABS(SIGX - SIGY).LT.D20) RETURN
      TAN2B = \{SIGXY \neq 2.0\}/\{SIGX - SIGY\}
      BETA = DATAN(TAN2B)*90./PI
Theta(j,2) = Beta
      RETURN
С
      AN I SOTROP IC
С
С
   80 CDA = 1.0/(1.0 - CNU12*CNU21)
      D(1,1) = E1 + CDA
      D{1,2} = E1 * CDA * CNU12
      D(2,1) = D(1,2)
      D(2,2) = E2*CDA
      DO 100 L = 1,2
      STEMP = 0.0
       DO 90 M = 1,2
   90 STEMP = STEMP + D(L, M)*EPR(J,M)
  100 \text{ STRP}(J_1L) = \text{STEMP}
      RETURN
C
С
      STEEL BAR STRAINS AND STRESSES
C
      BOND LINK STRAINS AND STRESSES
С
  110 XJI = X(JJ2) - X(JJ1)
      YJI = Y\{JJ2\} - Y\{JJ1\}
      SL2 = XJI*XJI + YJI*YJI
      KS1 = 2 \times ITOP(J,5)
      KS2 = 2 \times ITOP(J_0)
      ESPS(J) = (XJI*(U(KS2-1) - U(KS1-1)) + YJI*(U(KS2) - U(KS1)))/SL2
       IF(E1.LT.D2D) CALL EXIT
С
       IF STEEL STRESS-STRAIN CURVE HORIZONTAL PROGRAM STOPS
      STR S(J) = E1+ESPS(J)
c
С
       DETERMINE TOTAL STEEL STRESS FROM STRESS-STRAIN CURVE
C
```

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NCURV = NCURVS - 1
     IE = NZS
     DO 120 K = 1, NCURV
     DE = 1.00 10
     IF(DABS(E1 - ES(K)).LT.D10) DE = ZERO
     IF(OE.EQ.ZERO) IE = K
 120 CUNTINUE
     ED = TESPS(J) + ESPS(J) - SES(IE_2)
     TSTRS(J+2) = SES(IE,1) + ED*(SES(IE+1,1) - SES(IE,1))/
    1(SES(IE+1,2) - SES(IE,2))
     BOND STRAINS AND STRESSES
С
     MTYP = 3
     IF(JJ2.GE.(JJ1 + NOD2)) GOTO 125
     VERTICAL BAR
     CO = 0.00+00
     SI = 1.00+00
     KDIR = 2
     GOTO 130
C.
     HORIZONTAL BAR
 125 CO = 1.00+00
     SI = 0.00+00
     KDIR = 1
 130 CONTINUE
     KLAST = 0
     KB1 = 2 + JCNL(ETOP(J,3),2)
     KB2 = 2 + ITOP(J_{1}5)
     L = 2 + J - KDIR + 1
 140 EPSB(L,1) = - CO + U(KB1-1) - SI + U(KB1) + CO + U(KB2-1) +
            SI # U(KB2)
    1
     EPSB(L,2) = SI+U(KB1-1) - CO+U(KB1) - SI+U(KB2-1) + CO+U(KB2)
     EP1 = TEPSB(L,1) + EPSB(L,1)
     CALL TYPE ( L. MTYP )
     STRB(L,1) = VKH + EPSB(L,1)
     STRB(L,2) = VKV * EPSB(L,2)
     IF LAST BAR ENCOUNTERED, PROCESS END LINK STRESSES
     IF ( J .NE. NUMBAR ) GO TO 170
     IF ( KLAST .EQ. 1 ) GO TO 170
     L = 2 + (J + 1) - KDIR + 1
     KB1 = 2 + JCNL(ITOP(J,4),2)
     KB2 = 2 + ITOP(J,6)
     KLAST = 1
     GO TO 140
 17C RETURN
     END
     SUBROUTINE RESOUT(ICHECK)
c
  • SUBROUT INE RÉSOUT PRINTS ALL RESULTS AFTER EACH LOAD INCREMENT
C . RESULTS APPEAR IN TABULAR FORM
```

c

С

c

C

С

C.

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r

-1

```
COMMON S(10,10),VKH,VKV
       COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI
       COMMON NN+NH+HA+NODV+NODH+NUNNOD+NREL+NTEL+NUMBAR+MAXNDF+HBAND
       COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, IS CALE, NDF, IAUTO
       COMMON NUMDE, NUMTOF, NIT, IT, NC UR VB, NZ8, NOD2
С
       COMMON/ELEM / X(650),Y(650),THETA(1152,2),U(1300),TU(1300)
       COMMON/ELEM1 / JTOP(288,4), THICK(288)
       COMMON/TYP / ITYPE(1152), ISTYPE(50), IBTYPE(100), KDIR
       COMMON/REINE / SAREAL 501, JCNL(650,2), ITOPL 50,61, NREINE, IS1, IS2
       COMMON/CONSTR/ STR(1152,3), STRP(1152,2), EPS(1152,3), EPR(1152,2)
       COMMEN/CONST1/ TSTR(1152,3), TEPS(1152,3)
       CUMMON/STLSTR/ ESPS( 50), STRS( 50), TESPS( 50), TSTRS( 50,2)
       COMMON/ BUND / EPSB(100,2), STRB(100,2), TEPSB(100,2), TSTRB(100,2)
       DIMENSION SUN(3), ROWSUM(3)
       DATA ZER0/0.0000/ , 04/4.0000/ , IEQUI/'EQUI'/
С
٤
       PRINT NODAL DISPLACEMENTS
C
       ILINE = 0
       WRITE(6,1000)
 1000 FORMAT(1H1.42(1H*)/43H * NUDAL DISPLACEMENTS
                                                                         *.,
     1/,1X,42(1H*),/}
       WRITE(6,1001)
 1001 FORMAT( * NUDE NO.
                                                 Y-DISPLACEMENT +/)
                             X-DISPLACEMENT
      K = 0
      00 10 NM = 1,NUMTDF .2
       ILINE = ILINE + 1
       K = K + 1
       WRITE(6.1002) K.TU(NM), TU(NM+1)
 1002 FORMAT (1x, 15, 8x, 1PD12.5, 8X, 1PD12.5)
       IF(ILINE.NE.55) GOTO 10
       WRITE(6,1003)
 1003 FORMAT (1H1,//)
       #RITE(6.1001)
       ILINE = 0
   10 CONTINUE
C.
C
      PRINT ITERATION INFORMATION
C
       WRITE(6.7200) IT
 7200 FORMAT(/, 1X, "ERROR LESS THAN TOLERANCE IN ITERATION NO ", 13./.1X."
     c
С
      PRINT STRESSES IN CONCRETE
· C
      WRITE(6,1004)
 1004 FORMAT(1H1,42(1H*)/43H * STRESSES AND STRAINS IN CONCRETE
                                                                        *,
     1/.1x.42(1H+)./)
      WRITE(6,1005)
 10C5 FORMATL' ELEM ND. SIGMA - X
                                         SIGMA - Y
                                                         TAU - XY
                                                                        51
                                              TYPE +, /, 12X, *EP SILON-X
                  SIGNA - 2
                                  THETA(1)
     1GMA - 1
     2 EPSILON-Y
                                      EPSILON-1
                       GAHMA-XY
                                                     EPS ILUN-2
                                                                     THETAL
     3211,/1
      ILINE = 0
      DO 20 NE = 1,NTEL
      ILINE = ILINE + I
      IFILTYPEINEL.EQ.11 GUE: 15
```

INPLICIT REAL+8(A-H, 0-2)

```
WRITE(6,1060) NE, STRP (NE, 1), STRP (NE, 2), THETA(NE, 1), ITYPE(NE),
      1EPR(NE,1), EPR(NE,2), THETA(NE,1)
  1060 FORMAT(1X,15,48X,2(1PD12.5,2X),1X,1PD12.5,2X,13,/,54X,2(1PD12.5,2X
      1),1X,1PD12.5)
       IF(ITYPE(NEJ.EQ.4) WRITE(6,1007)
  1007 FORMAT(1H+,105X, *CRACKED*)
       GOTC 16
    15 WRITE(6,1006) NE, TSTR(NE, 1), TSTR(NE, 2), TSTR(NE, 3), STRP(NE, 1), STRP(
      INE, 2), THETA(NE, 2), ITYPE(NE), TEPS(NE, 1), TEPS(NE, 2), TEPS(NE, 3), EPR(N
      2E,1), EPRINE,2), THE TAINE,1)
 1006 FURMAT(1X, 15, 4X, 3(1PD12.5, 2X), 2X, 2(1PD12.5, 2X), 1X, 1PD12.5, 2X, 13,/,
      11x,5x,4x,3(1PD12.5,2x),2x,2(1PD12.5,2x),1x,1PD12.5)
    16 IF(ILINE.NE.28) GOTO 20
       #RITE(6,1003)
       #RITE(6,1005)
       ILINE = 0
   20 CONTINUE
       IFENREINF. EQ.01 GOTO 50
£
c
       PRINT STEEL AND BOND STRAINS AND STRESSES
٢
       WRITE(6.9070)
 9070 FORMAT(1H1,42(1H*),/43H * STRESSES & STRAINS IN REINFORCEMENTS *
      1,/,LX,42(1H+),/)
       WRITE(6,9080)
 9080 FORMAT (* BAR NO.
                           EPS IL ON
                                              SIGNA CALC.
                                                                 SIGMA DEP
      1ICTED*,5X,*TYPE*/)
       DO 30 IB = 1. NUMBAR
   30 WRITE(6,9090) IB, TESPS(IB), TSTRS(IB, 1), TSTRS(IB, 2), ISTYPE(IB)
 9090 FORMAT(15,5x,1PD12.5,6x,1PD12.5,5x,1PD12.5,10x,13)
       WRITE(6.9100)
 9100 FORMAT(1H1,42(1H*),/43H * BOND STRESSES
     1./.43H * IN DIRECTION OF GLOBAL AXES
                                                        *;/;1X;42(1H#};/)
      WRITE (6,9110)
       00 40 L = 1, NUMBAR
      K = 2 + i
       WRITE(6,9120) ITOP(L,5), TEPSB(K,1), TEPSB(K,2), TSTRB(K,1),
                     TSTRB(K.2), IBTYPE(K)
     1
       IF ( IBTYPE(L) .EQ. 3 ) WRITE(6,9030)
   40 CUNTINUE
      K = Z + ( NUMBAR + 1 )
      WRITE(6,9120) ITOP(NUMBAR,6), TEPSB(K,1), TEPSB(K,2), TSTRB(K,1),
                     TSTRB(K,2), IBTYPE(K)
     1
      IF ( IBTYPE(NUMBAR+1) .EQ. 3 ) WRITE(6,9030)
 9110 FURMAT (' LINK NO. AT NODE REL. X-DISP.
                                                     REL. Y-UISP.
                                                                       ¥ -
     IFORCE Y - FORCE - 3X, TYPE //
 9120 FORMAT ( 11x, 15, 5x, 4 / 19012.5, 3x ) ,12)
 9030 FORMAT(1H+,99X,*FAILED')
   5C IF(ICHECK.NE.IEQUI) RETURN
C
c
      STATIC EQUILIBRIUM CHECK
٤
      WRITE(6,9130)
 9130 FORMAT(1H1,42(1H*),/43H * EWUILIBRIUM CHECK FUR EACH CULUNN
     1,/,1X,42(1H*),/)
      WRITE(6,9135)
 9135 FORMATCIX," RECT. ELIN.
                                  AV. X-FURCE AV. Y FURCE AV. SHEAR FO
     INCE 1/1
      KP = 0
```

```
NHOR = NODH - 1
       NVER = NODV - 1
      1PAGE = 48/(NVER+3)
 9160 FORNAT(1H1,1X, " RECT. ELEM.
                                       AV. X-FORCE
                                                      AV. Y-FORCE
                                                                    AV. SHEA
     1R FORCE!/)
      IEL = 0
      DO 80 K = 1. NHOR
       DO 55 KK = 1.3
   55 ROWSUN(KK) = ZERO
      DO 70 KROW = 1, NVER
       DO 60 KK = 1,3
   60 SUM(KK) = ZERO
       IEL = IEL + 1
      ITEL = 4*IEL - 3
      D0.65 L = 1.4
       DO 66 LL = 1.3
   66 SUN(LL) = SUM(LL) + TSTR(ITEL,LL)
   65 ITEL = ITEL + 1
       DO 67 LL = 1.3
       SUM(LL) = SUM(LL)*(Y(KROW+1) - Y(KROW))*THICK(IEL)/04
   67 ROWSUM(LL) = ROWSUM(LL) + SUM(LL)
   70 WRITE(6,9140) IEL, (SUM(LL), LL=1,3)
       WRITE(6,9150) K, (ROWSUM(LL), LL=1,3)
      IF(KP+LT+IPAGE) GOTO 80
      KP = 0
      IPAGE = 54/(NVER+3)
      WRITE(6,9160)
   80 \text{ KP} = \text{KP} + 1
 9140 FORMAT( 16. 9X. 3( 2X. 1PD 12.5) )
 9150 FORMAT(/* ROW*, 15, 4X, 3(2X, 1PD12.5), * RESIDUAL FORCES*/)
      RETURN
      END
      SUBBOUT INE PRERI (J.)
      THIS SOUBROUTINE DECLARES ALL FATAL ERRORS
С
      PROGRAM STOPS AFTER ERROR MESSAGE IS PRINTED
      IMPLICIT REAL +8(A-H, 0-Z)
      COMMON S(10,10), VKH, VKV
      COMMON DXX, DYY, E1, E2, CNU, CNU12, CNU21, EP1, EP2, SNU, PI
      COMMON NN, NH, MA, NODV, NO DH, NUMNOD, NREL, NTEL, NUMBAR, MAXNDF, MBAND
      COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO
      COMMON NUMDE, NUMTOF, NIT, IT, NC UR VB, NZB, NUD2
С
      GOTO(1,2,3,4,5,6,7,8,9) , J
    1 PRINT 11
   11 FORMAT (///, 1X, 11HFATAL ERROR, //, 1X, 35HNUMBER OF NODES HORIZONTAL
     1IS ZERO)
      GOTO 99
    2 PRINT 12
   12 FORMAT(///,1X,11HFATAL ERROR,//1X,33HNUMBER OF NODES VERTICAL IS
     1ZEROJ
      GOTC 99
    3 PRINT 13
   13 FORMAT (///,1x,11HFATAL ERROR,//1x, 'NBEG IS EQUAL TO NEND')
      GOTO 99
    4 PRINT 14
   14 FORMAT(///,1x,11HFATAL ERROR,//1x, *STEEL AREA NEGATIVE*)
```

C

C.

£

```
GOTO 99
    5 PRINT 15
   15 FORMAT(///.1X.11HFATAL ERROR.//1X. INCORRECT ENDNODE)
      GOT 0 99
    6 PRINT 16
   16 FORMAT(///.1X.11HFATAL ERRUR.//1X. NUMBER OF REINFORCEMENTS SPECIF
     1 IED IS NEGATIVE")
      GO TO 99
    7 DR INT 17
   17 FORMAT(///,1X,11HFATAL ERROR,//1X,*STATEMENT 1 BEFORE STATEMENT 40
     14 IS FALSE , CALL UN PROGRAMMER*)
      GOTO 99
    8 PRINT 18 , MBAND
   18 FORMAT(///.1X.11HFATAL ERROR.//1X."BANDWIDTH EXEEDED MBAND =".[4."
         CALL PROGRAMMER!)
     1
      GO TO 99
    9 PRINT 19
   19 FORMAT (/// +1X +11 HEATAL ERROR +// +1X + BOUN DARY CONDITIONS ARE INCORR
     1ECTLY LABELED*)
   99 CALL EX IT
      RETURN
      END
      SUBROUTINE PRER2(J)
С
С
      THIS SUBROUTINE PRINTS NONFATAL ERROR MESSAGES
      THE PROGRAM CONTINUES AFTER ASSIGNING A VALUE TO THE VARIABLE
C
      WHICH IS OUT OF RANGE
C
r
      IMPLICIT REAL#8(A-H+0-Z)
      COMMON S(10,10), VKH, VKV
      COMMON DXX. DYY. E1. E2. CNU. CNU12. CNU21. EP1. EP2. SNU. PI
      COMMON NN, NH, MA, NODV, NODH, NUHNOD, NREL, NTEL, NUMBAR, MAXNDF, MBAND
      COMMON NINCR, KINCR, NCURVC, NCURVS, NZC, NZS, ISCALE, NDF, IAUTO
      COMMON NUMDE, NUMTOF, NIT, IT, NC UR VB, NZB, NOD2
      COMMON/MODULC/ SEC(10,2),EC(9),XNUC(9),TOL
      COMMON/MODULS/ SES(20 .2) .ES(19) .XNUS(19)
      COMMON/MODULB/ SEB(10,2),EB(9)
      PRINT 99
   59 FORMAT(/)
      PRINT 100
  100 FORMAT(1X, **** SUBROUTINE PRER2
                                                             ",14X,"***",/,
     11X, **** NONFATAL ERROR, SOLUTION CUNTINUES
                                                                    ***!)
      GOTO(1,2,3,4,5) , J
    1 PRINT 101,E1,E2
  101 FORMAT(/,1X, **** EMODUL OUT OF RANGE, LAST VALUES*, 19X, *****, T40, *
     1E1 =*,1PD12.4,/,T40,*E2 =*,1PD12.4)
      GOTO 999
    2 PRINT 101, E1, E2
      GUT 0 999
    3 PRINT 103
                                                                    ****,/,
  103 FORMAT(//,1X,**** NUMBER OF INCREMENTS SPECIFIED IS LESS
     11X, **** THAN ONE, ROGRAM ASSUMES NINCR = 1.
                                                                    ****}
      GO TO 999
    4 PRINT 104
  104 FORMAT(/.1X.**** TOTAL LOAD APPLIED DOES NOT GIVE RISE*,9X.*****/,
     11X, **** TO INELASTIC BEHAVIOR, SOLUTION ELASTIC ONLY
                                                                    ****1
      GOTO 999
    5 PRINT 105
  105 FURMAT(//,1X,**** NUMBER OF START NODE GREATER THAN NUMBER ****,/,
```

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### APPENDIX C

## INPUT SEQUENCE FOR NARCOS-2

1. Number of Problems:

one card: (I5)

 $\frac{1}{2}$ 

NPROB = number of problems

2. Problem Identification Card:

one card: (18A4)

NAME = problem title

3. First Control Card:

one card: (4I5)

	1	56	10 11	15 16	20		
{	NNV	NNH	KTC	P NR	EINF		
	NNV =	number o	f nodes ve	rtically			
		NNV > 0:	unequal s	pacing			
		NNV < 0:	equal spa	cing			
	NNH =	number of	nodes hor	rizontally			
		NNH > 0:	unequal s	pacing			
		NNH < 0:	equal spa	cing			
	KTOP	= input mo	ode param	eter			
		KTOP =	1: autom	atic mesh	generati	on	
		KTOP =	2: individ	lual input			
	NREIN	F = numbe	er of reinfo	orcements			

4. Coordinate Data Cards:

min. two cards: (I5, D12.4)

J DTEMP J DTEMP J = node number DTEMP = x- or y-coordinate of nodes for NNV < 0 or NNH < 0: one card, respectively for NNV > 0: NNV cards for NNH > 0: NNH cards

5. Second Control Card:

one card: (4I5)

15 610 1115 1620NTHNCURVCNCURVSNCURVBNTH = number of elements with irregular thicknessNCURVC = number of points on concrete curveNCURVS = number of points on steel curveNCURVB = number of points on steel curveNCURVB = number of points on bond curveforNTH = 0-all elements are of unit thickness

6. <u>Thickness (Optional)</u>:

NTH cards: (I5, D12.4)

1	5	6	17	
$\Lambda$				Ţ
1 ]	$\mathbf{NT}$	THICK		I

NT = element number (rectangle)

NT = 0: all elements are changed to new thickness

NT > 0: supply NTH cards

7. Stress-Strain Data:

8.

NCURVB cards: (I5, D12.4, D12.6)

15 617 1829ISTRESSSTRAINI = number of point on curveSTRESS = stress at point ISTRAIN = strain at point IPoisson's ratios:(NCURVB - 1) cards: (I5, D12.6)15 615 61INUBI = interval (between two points given under 7)NUB = Poisson's ratioBlocks 7 and 8 are repeated for steel and bond if the parameter

NREINF is greater than zero.

9. Bond Stiffness (Optional):

one card: (5X, D12.5)

VKV = bond stiffness in direction perpendicular to reinforcement This card must be omitted if NREINF = 0. If a blank card is supplied, VKV is chosen as mentioned in Chapter IV.

10. <u>Reinforcement (Optional)</u>:

NREINF cards: (215, D12.4, 215)



11. Tolerance:

one card: (5X, D12.5)

1 5 6 17 TOL

#### 12. Third Control Card:

one card: (4I5) 1 5 6 10 11 15 16 20 NLOAD NBCS NINCR NIT NLOAD = number of loads NBCS = number of boundary conditions NINCR = number of increments NINCR = 0: elastic solution NINCR = 1: automatic scaling

#### NINCR > 1: specified increments

#### NIT = number of iterations

13. Loading Data:

NLOAD cards: (I5, 2D12.4)

I = joint number

X = X-component of load at I

Y = Y-component of load at I

14. Boundary Conditions:

NBCS cards: (I5, 4X, A1, D12.4)

I = restrained node

**\***= X: X-restraint

\* = Y: Y-restraint

VALUE = value of prescribed displacement

15. Equilibrium Check:

one card: (A4)

**P** = alphanumeric parameter

**P= EQUI:** initializes checking procedure

A blank card is required if no checking is requested.

#### 16. <u>Remarks</u>:

- a. For NINCR > 1 group 13 is repeated NINCR-1 times.
- b. If during the increments 2, 3, . . ., etc., the boundary conditions remain the same, one blank card may be supplied instead of the whole block 14.
- c. If the EQUI card is inserted, the checks are done for all increments.

#### APPENDIX D

#### SAMPLE INPUT AND OUTPUT



	451
	442 459
+-	
	441 458
	440 457
	439 456
	438 455
	437 454
	436 453
	435 452
	443 400

a) ORIGINAL



b) MODIFIED TO INCLUDE STEEL NODES



•

.

SIMPLE BEAM. SCORE	ELIS CHECK	PROBLEM9 1	REINE	468 NODES	A 864 TRIANG
3 -28 1	1	1.0022.07 1		400 10020	
1 3 5•0					
9 22.0					•
468 81.0					
1 4 5	3				
12.0					
1 -4000.0	-0.002				
2 -3000.0	-0.001				
4 300+0	0.0001			<b>*</b>	
1 0.15	0.0001				
2 0.15					
3 0.15		•	· · · · ·		
1 -40000.0	-0.002				,
2 -30000.0	-0.001				
3 0.0	0•0	•			
4 30000.0	0.001				•
5 40000.0	0.002		•		
2 0.5					
4 0.3					
1 -2200-0	-0.001				
2	00001				
3 2200.0	0.001				
1.00000D.12		,			
2 9	2	444			
0.05					
	3				
145	-7000.0				
1 A 2 Y					
2 ^ 3 X					
4 X					
5 X					
6 X					
7 X					
8 X					
9 X		/	•		
EQUILIBRIUM CHECK					
145	-200-0				
	20000				
145	-200.0	,			
145	-200.0				
		· •			
145	-200.0				
• • •					
145	-200.0	e			
1:45	-200-0				
	~20000				
					4

-8•33333 -4•16667
-16.66667 -33.33333 -33.33333 -33.33333 -33.33333 -33.33333 -16.666667
-83.33333 -166.66667 -166.66667 -166.66667 -166.66667 -166.66667 -83.33333
 -83.33333 -166.66667 -166.66667 -166.66667 -166.66667 -166.66667 -83.33333

	1										
	SIMPLE	BEAM	APPROX	IMATION	,637	NODES	WITH	STEEL,	PMAX	= 3000	LBS
	-7	-46	1	1							
~	1		12.0								
	40	7	90.0 5	3							
	ō	. 8		2		1					
	1	-3000	0.0	00	2						
	2	-2980	•0	00	175						
	3	-2760	0.0	~•00	125						
	4 5	-1500	0.0	00	05						
	6	1) U U	.0	0.0	00						
	7	300	.0	0.00	01						
	1	• 1	.5								
	2	• 1	.5								
	د 4	• 1	.) 5								
	5	.1	.5								
	6	• 1	.5								
	1	-50000	0.0	00	5						
	2	-47000	•••	00	162						
	 	47000		0.00	162						
	5	50000	0	0.00	5						
	1	• 3	33333		-						
	2	·•3	33333								
	3.	• 3	33333								
	4	+2200	1-0	00	1						
	2	2200	•0	0.0	1						
	3	2200	0.0	0.00	1						
		1.0000	00D+12	• •							
	Ţ	0	05	1.0	2 5	74					
	1	8	5	3							
	1		-	-62.5							
	2			-125.							
	3			-125.							
	4			-125.							
	6			-125.							
	7		•	-62.5							
	1	X									
	2	X									
	5	x.									
	5	x									
	6	х									
	570	X									
	FQUI	. т									
	1			-4.1	6667						
	2			-8•3	3333						
	3			-8•3	3333						
	45			-8.3	2222						
	2			-000							
			• •								
					· · ·						

145 -200.0

145 -200•0
ELEM NO.	SIGMA - X Epsilon-X	SIGMA - Y EPSILON-Y	TAU - XY Gamma-Xy	SIGMA - 1 Epsilon-1	SIGMA - 2 Epsilon-2	THETA(1) THETA(2)	TYPE
2 5 3	-4.03533D 02	-8.92756D 00	-3.03160D 01	-6.61209D 00	-4.05849D 02	4.36765D 00 4.36765D 00	1
2 54	-4.64396D 02	-8.69907D 01	-8.12503D 01	-7.02419D 01 6.43287D-07	-4.811450 02	1.16477D 01 1.16477D 01	1
255	-3.51229D 02 -1.06346D-04	-2.14609D 02 -5.39749D-05	-8.27862D 01	-1.75589D 02 -3.90171D-05	-3.90249D 02 -1.21304D-04	2.52363D 01 2.52363D 01	1
256	-2.90366D 02 -8.99614D-05	-1.36546D 02 -3.09970D-05	-3.18519D 01 -2.44198D-05	-1.30211D 02 -2.85687D-05	-2.96701D 02 -9.23897D-05	1.12483D 01 1.12483D 01	1
257	2.98416D 02 9.94673D-05	9.14862D-02 -1.48903D-05	1.50205D 01 1.15157D-05	2.99170D 02 9.97565D-05	-6.62883D-01 -1.51795D-05	2.87513D 00 2.87513D 00	1
258	2.62184D 02 8.68555D-05	1.07839D 01 -9.51456D-06	1.12352D 00 8.61363D-07	2.62189D 02 8.68574D-05	1.07789D 01 -9.51648D-06	2.56050D-01 2.56050D-01	1
259				0.0 1.00842D-04	-4.54758D 01 -1.51586D-05	-2.75066D 00 -2.75066D 00	CF
260	2 200410 02	- 4 202540 00	3 193760 01	1.13187D-04	-2.02676D-05	-1.00132D-01 -1.00132D-01 7.94635D 00	7 CF
201	7.35621D-05	-1,24362D-05	2.44854D-05	7.52710D-05	-1.41451D-05	7.94635D 00 5.70504D 00	• . 1
262	6.02238D-05	-6.46157D-06	1.34583D-05 3.10479D 00	6.08960D-05 2.20004D 02	-7.13382D-06 -3.98053D 00	5.70504D 00 7.94314D-01	1
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	-2.62850D-05	-1.569970-05	2.457430-05	-7.613800-06	-3.437090-05	-3.334820 01	

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### VITA

# Alexander Jean Lassker

#### Candidate for the Degree of

#### Doctor of Philosophy

## Thesis: NONLINEAR BEHAVIOR OF REINFORCED CONCRETE BEAMS BY THE FINITE ELEMENT METHOD

#### Major Field: Civil Engineering

Biographical:

- Personal Data: Born in Arbon, Switzerland, May 11, 1940, son of Mr. and Mrs. Jean Paul Lassker.
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- Professional Experience: Research assistant, Swiss Federal Institute of Technology, 1965-1967; graduate teaching assistant, Oklahoma State University, 1967-1968; Consulting Engineer at the Computing Center of Fides, Inc., Zurich, Switzerland, summer, 1968; teaching assistant, Oklahoma State University, 1968-1971; Senior Engineer with R. R. Nicolet and Associates, Consulting Engineers, Montreal, Canada, 1971.