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[^0]
## ANGULAR FUNCTIONS BY

ELECTRONIC COMPUTER

## THESIS APPROVED:



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## NOMENCLATURE

```
    A - cross sectional area
    b - width of plate
    E - modulus of elasticity
    F
    GAB, GBA - angular carry-over
        h - depth of plate
        I - moment of inertia
        L. - length of span
Li
Mi, M j, M M - bending moments at points i, j, and k
        m - number of plates
        n - number of segments
        P - intensity of concentrated load
        P
        S - section modulus
        w - intensity of load
        DL - dead load
        UL - uniform load
        \Delta-deflection
        e-density of construction material
\tau
```


## INTRODUCTION

## 1-1 Discussion

The analysis of a continuous plate girder structure may be solved by either the flexibility or stiffness approach. In this thesis the flexibility method is considered with the three moment equation adopted as the mathematical model for analysis.

Computing angular functions for a structure which possesses numerous cross sectional variations, by conventional methods, is a laborious process. Much current literature is advocating the use of electronic computers for structural problems which are of a repetitive nature. The primary benefits obtained are economy, increase in productivity, solutions with greater accuracy, and optimization of design.

Many highway departments have developed computer programs to assist in the analysis and design of simple and continuous bridges. A series of four reports on the analysis of continuous beam bridges published by the School of Civil Engineering at Oklahoma State University $(4,7,8,10)$ is concerned with the complete analysis of a parabolic haunched bridge by electronic computer. Exline (3) presented a computer program for angular functions for a member with abrupt change in cross section but limited to members with a maximum of three variations.

In this thesis three separate programs for the IBM 650 electronic computer are developed in FORTRAN language utilizing floating point arithmetic. These programs are for sectional properties, angular functions, and deflections due to dead or uniform loading.

## 1-2_Flexibility Method

Having calculated the angular flexibilities, angular carry-over functions, and angular load functions for a structure of variable cross section by electronic computer, the three moment equation flexibility matrix is obtained by substitution. By inverting the flexibility matrix, a solution for final end moments may be obtained.

The three moment equation

$$
G_{i j} M_{i}+\sum F_{j} M_{j}+G_{k j} M_{k}=-\sum \tau_{j}
$$

may conveninetly be written for each redundant and placed in matrix form.

Using abbreviated notation, equation (1) can be rewritten

$$
\begin{equation*}
\left[F_{j}\right]\left\{M_{j}\right\}=(-1) \quad\left\{\Sigma \tau_{j}\right\} \tag{2}
\end{equation*}
$$

where:

$$
\left[F_{j}\right] \text { - flexibility matrix }
$$

Inversion of the flexibility matrix yields a solution for final
end moments.

$$
\left\{M_{j}\right\}=-\left[F_{j}\right]^{-1}\left\{\begin{array}{lll} 
& \tau &  \tag{3}\\
& & j
\end{array}\right.
$$

Programs for matrix inversions are readily available; thus, they have not been developed in this thesis. Once final end moments are known, final deflections may be obtained as developed in Chapter IV.

CHAPTER II

CROSS SECTIONAL PROPERTIES

## 2-1 Discussion

Determination of the sectional properties of each unique cross section along the span is necessary for the evaluation of angular functions. Those properties considered and programed include:
$A_{X}$ - cross-sectional area at section $X$
$I_{x}$ - moment of inertia at section $x$
$h_{x}$ - total depth at section $x$
$S_{T x}$ - section modulus for top at section $x$
$S_{B x}$ - section modulus for bottom at section $x$
The moment of inertia $I_{x}$ is used directly in the evaluation of angular functions (Chapter III). The cross sectional area $A_{X}$ is needed in the evaluation of angular functions for dead load. Although the application of the sectional moduli $\mathrm{S}_{\mathrm{Tx}}$ and $\mathrm{S}_{\mathrm{Bx}}$ is beyond the scope of this thesis, they are easily obtained and are included for use in the possible extension of this thesis. The total depth of each section is also included for the designer's convenience.

## 2-2 Derivation

A typical span $A B$ (Fig. 2-1) of a continuous plate-girder of variable cross section is divided into n segments. For segments


Typical Span $A B$ of $n$ Unequal Strips


Figure 2-2
Typical Cross Section With Centroid Location
of uniform cross section each change in cross section may be considered the termination and beginning of a segment. The width and depth of a typical plate at section $x$ is designated as $b_{x j}$ and $h_{x j}$ respectively (Fig. 2-2). The width and depth of a segment of nonuniform cross section must be approximated by some appropriate method. For example, the depth of a segment whose depth is not constant may be approximated as the depth at mid-segment or as an average of the end depths. Numbering of sections begins with two to more easily facilitate writing of the computer program for Phase II (Angular Functions). Each time a subscript designates a particular segment, the cross section to the left of the subscript is inferred.

The cross sectional area of segment $x$ is

$$
\begin{equation*}
A_{x}=\sum_{j=1}^{m} b_{x j} h_{x j} \tag{2-1}
\end{equation*}
$$

The distance from the bottom plate to the centroid of the section (Fig. 2-2) may be evaluated as

$$
\begin{equation*}
\bar{Y}_{B x}=\frac{\sum_{j=1}^{m}\left(b_{x j} h_{x j}\right)\left(\bar{Y}_{B x, j}\right)}{\sum_{j=1}^{m} b_{x j} h_{x j}} \tag{2-2}
\end{equation*}
$$

The distance from the top plate to the centroid of the section is

$$
\begin{equation*}
\overline{\mathbb{T}}_{\mathrm{T} X}=\sum_{\mathrm{j}=1}^{\mathrm{m}}\left(\mathrm{~h}_{\mathrm{xj}}\right)-\overline{\mathrm{F}}_{\mathrm{Bx}} \tag{2-3}
\end{equation*}
$$

The distance from the centroid of the section to the centroid of each plate is

$$
\begin{equation*}
\bar{Y}_{x j}=\bar{Y}_{B x}-\bar{Y}_{B x, j} \tag{2-4}
\end{equation*}
$$

The moment of inertia of a typical section is

$$
\begin{equation*}
I_{x}=\sum_{j=1}^{m}\left[\frac{b_{x j} h_{x j}}{12}+b_{x j} h_{x j} \bar{Y}_{x j}^{2}\right] \tag{2-5}
\end{equation*}
$$

And the corresponding section moduli are

$$
\begin{equation*}
\mathrm{S}_{\mathrm{Tx}}=\frac{\mathrm{I}_{\mathrm{x}}}{\overline{\mathrm{Y}}_{\mathrm{Tx}}} \tag{2-6}
\end{equation*}
$$

and

## 2-3 Computer Program

## 2-3-1 Results of Program

The program presented (Fig's. 2-3,4) is written for the IBM 650 electronic computer using FORTRAN language and floating point arithmetic. Notation of typical section $x$ in the derivation has been changed to section $i$ in the computer program. The program yields a solution for:

$$
\begin{aligned}
& \operatorname{AREA}(I) \text { - the cross sectional area of section } i \\
& \operatorname{EYE}(I) \text { - the moment of inertia of section } i
\end{aligned}
$$

SXTOP(I) - the section modulus for the top for section i SXBOT(I) - the section modulus for the bottom for section i DEPTH(I) - the total depth of section i

## 2-3-2 Data Required

The input data necessary for computing the desired sectional properties include:

```
            N - the total number of strip segments
        M(I) - the total number of plates at section i
    B(I,J) - the width of plate j at section i
    H(I,J) - the depth of plate j at section i
```


## 2-3-3 Auxiliary Quantities

Other quantities integral in the program are denoted as:
YBOT(I) - the distance from the bottom to the centroid of section i

YTOP(I) - the distance from the top to the centroid of section i
$\operatorname{YBAR}(I, J)$ - the distance from the bottom of section $i$ to the centroid of plate $j$

YPLAT(I,J) - the distance from the centroid of section $i$ to the centroid of plate $j$


Flow Chart Phase I - Sectional Properties

```
        C 0000 O PHASE ONE-EVALUATION OF CROSS
    C 0000 0 SECTIONAL PROPERTIES AREA,
    OOOO O MOMENT OF INERTIA,DEPTH,AND
    C OOOO O SECTION MODULI
    1 O DIMENSION AREA(25), EYE(25).
        1. 1 YBOT(25),YIOP(25), SXTOP(25i,
        1:1 YBOT(25);YTOP(25), SXTOP(25
        1 2 SXBOT(25), M(25): B{25,9)
    l00 O DIMENSIONH(25,9): YBAR(
    1002 TYBOT(25)
        2 O READ, N
        30.00 103 l=20N
        40 READ, MII
    101 O L = M\I)
    O O DO 103 J#1/L
    103 O READ, B(I.J), H(l,J)
C OOOO'O CALCULATE CROSS SECTIONAL AREA
        5 O DO 9 I=2,N
        6 AREA (I) =0.0
        70L=M\I!
        8 0 ~ D O ~ 9 ~ J = 1 O L
        90 AREAII
C 0000 O CALCULATE DISTANCE FROM
C 0000 1 BOTTOM TO CENTROID OF
    O00 2 EACH PLATE - YBAR
        10 O DO 14 I=2ON
        1! 0 YBAR(I,1)= H(I,1)/2.0
        12OL=M(I)
        130 DO 14 J=2:L
        140 YBARII:JI= YBAR(I:J-1)
        14 l + H(I.J.-1)/2.0 + H(I:J)/2.0
C 0000 O CALCULATE DISTANCE FROM
    0000 O BOTTOM TO CENTROID OF SECTION
    15ODO 16. I =2IN
    16 0 TYBOT(I)= =0.0
    180L=M(1)
    190 DO 20 J=1,L
    200 TYBOT(l) = (TYBOT(I)) +
        201(IB(I;J)*H(I;J))* YSAR(I;J)
        210 YBOT\It = TYBOT\I!/AREAII)
C O000 O CALCULATE DISTANCE FROM TOP
    O000 O TO CENTROID OF EACH SECTION
    220 DO 27 I=2.N
    230 DEPTH(I) = 0.0
    240L=M(I)
    25 0 DO 26 J=1,L
    25 0 DEPPTH(I) = DEPTH(I) +H(I.J)
    260 DEPTH(I) = DEPTH(I) + H(I*J)
C 0000 O CALCULATE DISTANCE FROM
C. 0000 2 CENTROID OF THE SECTION
C. 0000 2 CENTROID OF THE SECTION
    2800 DO 3I I =2,N
    30 0 DO 31 J=1,
    30 0 DO 31 J=1:L M YBOT(II) -
    311 YBAR(I,J) = YBOT(II
C 0000 O CALCULATE MOMENT OF
    320}
    32O DO 36 I=2,N
    330 EYEIIJ = O.O
    34 O L =M(I)
    35 O DO 36 J=1,L
    36 O EYE(I) = EYE(I) + (BI!,J)*
    36 1 (H(I,J)**30I)/ 12*+ + (I|J)*
C 0000 2 H![&J) * (YPLAT(I,J)**2.1
C 0000 1 CALCULATE THE SECTION MODULI
    370. DOTOP AND SXBOT
    0. DO 4l l=2,N
    380 SXTOP(I)= EYE(1)/YTOP(I)
C OOO O SXBOTII) = EYEIII/YBOTIII
    O PUNCH RESULTS
    40 0 PUNCH, EYEIII, AREA(I)
    41 O PUNCH, DEPTH(II: SXTOP(I),
    41 5 SXBOTIII
    420 GO TO 2
    Figure 2-4
FORTRAE PROGRAM - Section Properties
```

ANGULAR FUNCTIONS

## 3-1 Discussion

Angular functions (flexibilities, carry-over flexibilities, and load functions) are and slopes due to unit cause or due to loads (Table 3-1). Numerous procedures are available for evaluating these functions:

1. Integral evaluation with unequal length segments.
2. Integral evaluation with equal length segments.
3. Conjugate beam method with unequal length segments.
4. Conjugate beam method with equal length segments.

It is the opinion of the author that an approach which yields not only the angular functions, but with additional steps will yield deflections, is desirable. One such method is that of the conjugate beam utilizing the string polygon (9). Advantages of using equal or unequal length strip segments are unique with each method. Equal length segments are perhaps easier to program and possess the advantage of allowing one to obtain deflections at equal spacings along the span. However, unequal length segments as determined by the cross sectional variation will yield a more accurate solution as compared with equal length segments, considering an equal number of segments to be used in both methods. When additional points of deflection

| TERM | NAME | VALUE | $\begin{gathered} \text { VALUE FOR } \\ \text { CROSSSSANT } \end{gathered}$ | PHYSICAL MEANING | ILLUSTRATION |
| :---: | :---: | :---: | :---: | :---: | :---: |
| $\mathrm{F}_{\mathrm{ji}}$ | ANGULAR FLEXIBILIT | $\int_{j}^{i} \frac{x^{2} d x}{L_{j}^{2} E I x}$ | $\frac{L_{j}}{3 E I_{j}}$ | end slope of a simple beam ij at $j$ due to a unit moment applied at that end | $M=1$ <br> (i) |
| $F_{j k}$ | ANGULAR FLEXIBILITY | $\int_{j}^{k} \frac{x^{\prime 2}}{L_{k}} Z^{2} E I_{x}$ | $\frac{\mathrm{L}_{\mathrm{k}}}{3 \mathrm{EI}}$ | end slope of a simple beam $j k$ at $j$ due to a unit moment applied at that end | $M=1$ <br> (1) <br> (k) |
| $G_{i j}$ | $\begin{aligned} & \text { CARRY-OVER } \\ & \text { FLEXIBILITY } \end{aligned}$ | $\int_{i}^{j} \frac{x x^{\prime} d x}{L_{j}^{2} E I x}$ | $\frac{L_{j}}{6 E I_{j}}$ | end slope of a simple beam ij at i due to a unit moment applied at the far end $j$ | $M=1$ <br> (i) |
| $\mathrm{G}_{\mathrm{kj}}$ | $\begin{aligned} & \text { CARRY-OVER } \\ & \text { FLEXIBILITY } \end{aligned}$ | $\int_{k}^{j} \frac{x x^{\prime} d x}{L_{k}^{2 E I} x}$ | $\frac{L_{k}}{6 E I_{k}}$ | end slope of a simple beam $j k$ at $k$ due to a unit moment applied at the far end $j$ |  |



Table 3-1
Interpretation of Angular Punctions
for the method of unequal segments are desired, auxiliary segments can be included. The author has thus chosen to use the conjugate beam approach utilizing the string polygon with unequal length segments.

## 3-2 Angular Flexibilities and Carry-Over Functions

## 3-2-1 Discussion

A typical span $A B$ of a continuous beam of several spans is considered (Fig. 3-1). Angular flexibility $F_{A B}$ and angular carryover $G_{B A}$ are obtained by applying a unit moment at A (Fig. 3-2). The reaction of the conjugate structure at $A$ is $F_{A B}$ and the reaction at $B$ is $G_{B A}$. Likewise, by applying a unit moment at $B$ on the real structure (Fig. 3-3), the reaction of the conjugate structure at $B$ is angular flexibility $\mathrm{F}_{\mathrm{BA}}$ and the reaction at A is angular carryover $G_{A B}$. By virtue of Maxwell's Reciprocal Theorem, the angular carry-over at $A$ is equal to the angular carry-over at $B\left(G_{A B}=G_{B A}\right)$.

Span $A B$ may be divided into $n$ unequal segments. Applying the string polygon method (9), elastic weights for each segment are evaluated and applied as loads to the conjugate structure.

A typical elastic weight is expressed in terms of the three moment equation

$$
\begin{equation*}
\bar{P}_{j}=M_{i} G_{i j}+M_{j} \sum F_{j}+M_{k} G_{k j}+\Sigma \tau_{j} \tag{3-1}
\end{equation*}
$$

‥ The moment at various ordinates $x$ due to either unit moment at $A$ or $B$ is a linear function of $X$.


Real Structure


$$
\begin{array}{ll}
M_{i}^{A}=\frac{x_{i}^{\prime}}{L} & M_{i}^{B}=\frac{x_{j}}{L} \\
M_{j}^{A}=\frac{x_{j}}{L} & M_{j}^{B}=\frac{x_{i}}{L}  \tag{3-2}\\
M_{k}^{A}=\frac{x_{k}}{L} & M_{k}^{B}=\frac{x_{k}}{L}
\end{array}
$$

and the end slopes due to loads are zero

$$
\sum \tau_{j}^{A}=0 \quad \sum \tau_{j}^{B}=0
$$

Thus, from eq. 3-1

$$
\begin{equation*}
\bar{P}_{j}^{A}=\frac{x_{i}^{\prime}}{L} G_{i j}+\frac{x_{j}^{\prime} \Sigma F_{j}}{\bar{L}}+\frac{x_{k}^{\prime} G_{k j}}{\bar{L}} \tag{3-3}
\end{equation*}
$$

and

$$
\begin{equation*}
\bar{P}_{j}^{B}=\frac{x_{i}}{L} G_{i j}+\frac{x_{j}}{L} \sum F_{j}+\frac{x_{k}}{L} G_{k j} \tag{3-4}
\end{equation*}
$$

where:

$$
\begin{aligned}
& \bar{P}_{j}^{A} \text { - elastic weight at section } j \text { due to a unit moment at } A \\
& \bar{P}_{j}^{B} \text { - elastic weight at section } j \text { due to a unit moment at } B \\
& G_{i j} \text { - angular carry-over flexibility of segment } i j \\
& G_{k j} \text { - angular carry-over flexibility of segment } k j \\
& \Sigma F_{j}=F_{j i}+F_{j k}-\begin{array}{l}
\text { sum of the angular } \\
\text { segments } j i \text { and } j k
\end{array}
\end{aligned}
$$

The conjugate reactions are then

$$
\begin{align*}
& F_{A B}=\sum_{j=1}^{n} \bar{P}_{j}^{A} \frac{x_{j}}{\mathrm{~L}} \\
& G_{B A}=\sum_{j=1}^{n} \bar{P}_{j}^{A} \frac{x_{j}}{L} \tag{3-5}
\end{align*}
$$

and

$$
\begin{align*}
& F_{B A}=\sum_{j=1}^{n} \bar{P}_{j}^{B} \frac{x_{j}}{I} \\
& G_{A B}=\sum_{j=1}^{n} \bar{P}_{j}^{B} \frac{x_{j}}{}{ }^{\prime} \tag{3-6}
\end{align*}
$$

## 3-2-2 Segment of Constant Cross Section

Considering typical segment ij (Fig. 3-4) of constant cross section, the following refinements may be made for angular flexibilities and carry-over functions (Eq's. 3-7).


Figure 3-4
Segment of Constant Cross Section

$$
\begin{equation*}
F_{i j}=F_{j i}=F_{j}=\frac{I_{j}}{3 E I_{j}} \tag{3-7}
\end{equation*}
$$

$$
G_{i j}=G_{j i}=G_{j}=\frac{L_{j}}{6 E_{j}}
$$

or,

$$
\begin{align*}
& F_{j}=\frac{L}{E I_{0}} \cdot \frac{L_{j} I_{0}}{3 L I_{j}}=\frac{L}{E I_{0}} F_{j}^{\prime} \\
& G_{j}=\frac{L}{E I_{0}} \cdot \frac{L_{j} I_{0}}{6 L I_{j}}=\frac{L}{E I_{0}} G_{j}^{\prime} \tag{3-8}
\end{align*}
$$

where:

$$
\begin{aligned}
& F_{j}{ }^{\prime}=\frac{L_{j} I_{0}}{3 L I_{j}} \\
& G_{j}^{\prime}=\frac{L_{j} I_{0}}{6 L I_{j}}
\end{aligned}
$$

This assumption is reasonably justified for sections whose cross section is not constant, provided average dimensions are taken for the plate dimensions.

From Eq's. 3-3, 4, the expression for a typical elastic weight due to unit moment at A or B respectively, is

$$
\begin{align*}
& \bar{P}_{j}^{A}=\frac{L}{E I_{0}}\left[\frac{x_{i}^{\prime}}{L} G_{j}^{\prime}+\frac{x_{j}^{\prime}}{L}\left(F_{j}^{\prime}+F_{k}^{\prime}\right)+\frac{x_{k}}{L} G_{k}^{\prime}\right]  \tag{3-9}\\
& \bar{P}_{j}^{B}=\frac{L}{E I_{0}}\left[\frac{x_{i}}{L} G_{j}^{\prime}+\frac{x_{j}}{L}\left(F_{j}^{\prime}+F_{k}^{\prime}\right)+\frac{x_{k}}{L} G_{k}^{\prime}\right]
\end{align*}
$$

From Eq's. 3-5, 6 , the end reactions of the conjugate beam become

$$
\begin{align*}
& F_{A B}=\frac{L}{E I_{0}} F_{A B}^{\prime} \\
& F_{B A}=\frac{L}{E I_{0}} F_{B A^{\prime}}^{\prime} \tag{3-10}
\end{align*}
$$

$$
G_{A B}=G_{B A}=\frac{L}{E I_{0}} G_{A B}^{\prime}
$$

## 3-3 Angular Load Functions

## 3-3-1 Discussion

By definition, angular load functions are end slopes due to loads (Table 3-1). Mathematically stated:

$$
\tau_{A B}=\int_{A}^{B} \frac{B M_{x} x^{\prime} d x}{L E I_{x}}
$$

$$
\tau_{B A}=\int_{A}^{B} \frac{B M_{x} x d x}{L E I_{x}}
$$

where:
$B M_{x}=$ the bending moment at section $x$ of the simple span
$A B$ due to loads.
Those cases of loading considered in this thesis include:

1. Unit load applied at each cut-off ordinate (live load)
2. Uniform load
3. Dead load

3-3-2 Unit Load
The angular load functions due to a unit load at each cut-off ordinate is useful not only for evaluating influence line ordinates for live loads, but also for evaluating the angular load functions due to uniform load and dead load.

The moment at any ordinate along span $A B$ due to a unit moment
at $A(o r B)$ is a linear function of that ordinate (Eq. 3-2).
The conjugate shear at $A, \bar{V}_{A B}$, (Fig. 3-6) due to a unit moment at $A$ or $B$, respectively, is

$$
\begin{aligned}
\overline{\mathrm{V}}_{A B}^{A} & =\sum_{j=1}^{n} \bar{P}_{j}^{A} \frac{x_{j}^{\prime}}{L} \\
& =\frac{L}{E I_{0}} F_{A B}^{\prime}
\end{aligned}
$$

and

$$
\begin{align*}
\bar{V}_{A B}^{B} & =\sum_{j=1}^{n} \bar{P}_{j}^{B} \frac{x_{j}^{\prime}}{L}  \tag{3-11}\\
& =\frac{L}{E I_{0}} G_{A B^{\prime}}
\end{align*}
$$

The conjugate shear at a typical ordinate, $\bar{V}_{j}$, (Fig. 3-6) due to a unit moment at $A$ or $B$, respectively, is

$$
\begin{aligned}
\overline{\mathrm{V}}_{j}^{A} & =\overline{\mathrm{V}}_{i}^{A}-\bar{P}_{i}^{A} \\
& =\frac{L}{E I_{0}} \overline{\mathrm{~V}}_{j}^{\mathrm{A}}
\end{aligned}
$$

and

$$
\begin{align*}
\bar{v}_{j}^{B} & =\bar{v}_{i}^{B}-\bar{P}_{i}^{B}  \tag{3-12}\\
& =\frac{L}{E I_{o}} \overline{\mathrm{~V}}_{j}^{B}
\end{align*}
$$

which by virtue of the theory of the conjugate structure is equal to the slope of the corresponding section of the real structure.

$$
\bar{V}_{j}^{A}=\theta_{j}^{A}
$$


(b)
moment diagram due to $M_{B}=1$

(c)
moment diagram due to $M_{A}=1$
Figure 3-5
Real Structure With Moment Diagrams Due To Unit End Moments

(c)
conjugate moment diagram
Figure 3-6
Conjugate Structure With Shear And Moment Diagrams

$$
\bar{v}_{j}^{B}=\Theta_{j}^{B}
$$

The conjugate bending moment at any ordinate, $\bar{M}_{j}$, (Fig. 3-6) due to a unit -moment at A or B, respectively, can then be expressed

$$
\begin{align*}
\bar{M}_{j}^{A} & =\bar{M}_{i}^{A}+\bar{V}_{j}^{A} I_{j} \\
& =\frac{L^{2}}{E I_{0}} \bar{M}_{j}^{A} \tag{3-13}
\end{align*}
$$

and
which by virtue of the theory of the conjugate structure is equal to the deflection of the corresponding section of the real structure. Thus,

$$
\begin{aligned}
\overline{\mathrm{M}}_{j}^{A} & =\Delta_{j}^{A} \\
\overline{\mathrm{M}}_{j}^{B} & =\Delta_{j}^{B}
\end{aligned}
$$

From Maxwell's Reciprocal Principle, the deflection at section j due to a unit moment at $A\left(M_{A}=1\right)$ is numerically equal to the end slope at $A$ due to a unit force at section $j$. Thus,

$$
\overline{\mathrm{M}}_{\mathrm{j}}^{\mathrm{A}}=\triangle_{j}^{A}=\left(\tau_{A B}\right)_{j}
$$

and

$$
\begin{equation*}
\bar{M}_{j}^{B}=\Delta_{j}^{B}=\left(\tau_{B A}\right)_{j} \tag{3-14}
\end{equation*}
$$

where: $\left(\tau_{A B}\right)_{j}-\underset{\text { ungular load function (end slope })}{ }$ unplied at at $A$ duection $j$ to a $\left(\mathcal{\tau}_{\mathrm{BA}}\right)_{j}$ - angular load function (end slope) at $B$ due to a 3-3-3 Uniform Load

Typical span $A B$ is loaded by a uniform load of intensity w (Fig. 3-7).


Figure 3-7
Uniform Loading

For a typical concentrated load $\mathrm{P}_{\mathrm{j}}^{\mathrm{UL}}$, one half the load of segments $j$ and $k$ are considered to be concentrated at $j$.

$$
P_{j}^{\mathrm{UL}}=w\left[\frac{L_{i}}{2}+\frac{L_{k}}{2}\right]
$$

The angular load function for uniform load is obtained by multiplying the equivalent concentrated loads by the corresponding end slope influence line ordinates (Eq's. 3-15).

$$
\tau_{A B}^{U L}=\sum_{j=1}^{n} P_{j}^{U L} \cdot\left(\tau_{A B}\right)_{j}
$$

$$
\begin{aligned}
& \tau_{A B}^{U L}=\frac{w I^{3}}{E I_{0}} \tau_{A B}^{U U^{\prime}} \\
& \tau_{B A}^{U L}=\sum_{j=1}^{n} P_{j}^{U L} \quad . \quad\left(\tau_{B A}\right)_{j} \\
& \tau_{B A}^{U L}=\frac{w I^{3}}{E I} \tau_{0}^{U L^{\prime}}
\end{aligned}
$$

where: w - uniform load intensity per unit length
$\tau_{A B}^{U L}$ - angular load function (end slope) at $A$ due to uniform
$\tau_{\mathrm{BA}}^{\mathrm{UL}}$ - angular load function (end slope) at $B$ due to uniform

## 3-3-4 Dead Load

Typical span $A B$ is loaded by dead load (Fig. 3-8). If typical segment $i j$ is considered to be of uniform cross section (average plate dimensions used for calculating cross sectional area), it may be considered to be loaded by a uniform load of magnitude $\mathrm{w}_{\mathrm{j}}$.


Figure 3-8
Dead Load

For a typical concentrated load $P_{j}$, one half the load of segments $j$ and $k$ are considered to be concentrated at $j$.

$$
P_{j}^{D L}=\rho A_{0}\left[\frac{A_{j}}{A_{0}} \frac{L_{j}}{2}+\frac{A_{k}}{A_{0}} \frac{L_{k}}{2}\right]
$$

The angular load function for dead load is obtained by multiplying the equivalent concentrated loads by the corresponding end slope influence line ordinates (Eq's. 3-16).

$$
\begin{align*}
\tau_{A B}^{D L} & =\sum_{j=1}^{n} P_{j}^{D L} \cdot\left(\tau_{A B}\right)_{j} \\
& =\frac{\rho_{0}^{A} L^{3}}{E I_{O}} \tau_{A B}^{D L^{\prime}} \\
\tau_{B A}^{D L} & =\sum_{j=1}^{n} \cdot P_{j}^{D L} \cdot\left(\tau_{B A}\right)_{j}  \tag{3-16}\\
& =\frac{\rho A_{O L^{\prime}}^{3}}{E I_{O}} \tau_{B A}^{D L \prime}
\end{align*}
$$

where: $\quad \rho$ - unit weight of construction material
$\tau_{A B}^{D L}$ - angular load function (end slope) at $A$ due to dead $\tau_{B A}^{D L}$ - angular load function (end slope) at $B$ due to dead $A_{0}$ - reference area

## 3-4 Computer Program

## 3-4-1 Results of Program

The Phase II computer program (Fig's. 3-7, 10) is written for the IBM 650 electronic computer using FORTRAN language and floating point arithmethic The program yields a solution for:

FABPM - angular flexibility coefficient at end $A$ of span $A B$
FBAPM - angular flexibility coefficient at end $B$ of span $A B$
GAORB - angular carry-over coefficient at either end $A$ or $B$ of span $A B$

BMA(I) - conjugate moment coefficient at section $i$ of the conjugate structure due to unit moment at $A\left(M_{A}=1\right)$; also the deflection coefficient of the real structure at $i$ due to unit moment at A; also the angular load function coefficient at A due to a unit force applied at section i
$B M B(I)$ - conjugate moment coefficient at section $i$ of the conjugate structure due to unit moment at $B\left(M_{B}=1\right)$; also the deflection coefficient of the real structure at i due to unit moment at B ; also the angular load function coefficient at A due to a unit force applied at section i

TABUL - angular load function coefficient at $A$ due to uniform loading

TBAUL - angular load function coefficient at $B$ due to uniform loading

TABDL - angular load function coefficient at A due to dead load
TBADL - angular load function coefficient at $B$ due to dead load

## 3-4-2 Data Required

The input data necessary for computing the desired angular functions include:

$$
N \text { - total number of segments }
$$

$S(I)$ - distance from left end of span $A B$ to the cut-off point at $i$ divided by the span length $L \frac{\left(x_{i}\right)}{L}$
EYE(I) - moment of inertia of segment i
AREA(I) - cross sectional area of segment i
The quantities $\operatorname{EYE}(I)$ and $\operatorname{AREA}(I)$ are read directly from the results of the section properties program (Phase I).

## 3-4-3 Auxiliary Quantities

Other quantities integral in the program are denoted:
FPRIM(I) - flexibility coefficient of segment i
GPRIM(I) - angular carry-over coefficient of segment i
PRIMA(I) - individual elastic weight coefficient at i due to a unit moment at $A\left(M_{A}=1\right)$

PRIMB(I) - individual elastic weight coefficient at i due to a unit moment at $B\left(M_{B}=1\right)$

VBARA(I) - conjugate shear coefficient at section $i$ of the conjugate structure due to unit moment at $A\left(M_{A}=1\right)$
$\operatorname{VBARB}(I)$ - conjugate shear coefficient at section $i$ of the conjugate structure due to unit moment at $B\left(M_{B}=1\right)$

EYEO - reference moment of inertia (taken as EYE(2) in computer program)

AREAO - reference area (taken as AREA(2) in computer program)


```
C.0000 0 PHASE 2
C 0000 O EVALUATION OF ANGULAR FUNCTION
C 0000 O ANGULAR FLEXIBILITIES AND
C OOOO O CARRY OVER VALUES
    1.0 DIMENSION S(25), EYE(25).
    1 FPRIM(25). GPRIM(25):
        2 PRIMA(25). PRIMB(25), AREA(25)
        3,VBARA(25),VBARB(25)
        1 4 BBMA(25): BMB(25)
        2 O READ, N
        30 READ, (S(I); I=1,N)
        40 DO 5 I =2,N
        50 REAO, EYE(I), AREA(II
        & O EYEO = EYE{2}
        AREAO = AREA(2)
    7.O DO 9.I=2oN
    80 FPRIM(I) (ISII) - S(I-I))*
    8 O FPRIM\I) (ISII) N
    8 1 EYEOI / (3.*EEYEII)!
    90 GPRIM(I
    110 DO 13 I =2,K
    12 0 PRIMB(I) =S(I-1)*GPRIM(I)
    121 +S(I) (FPRIM(I) + FPRIM(I+1
    121 +SIII #{FPRIM(I) + FPRIM
    12 }2|+S(I+1)*GPRIM(I+I
    13 O PRIMAII) = (1.-S(I-1)| *GPRIM
    131 (I) + (1.-5(II) (FPRIM{I) +
    132 FPRIM(1+1)) +11.-5(I+1)\*
    13 GPRIM(1+1)
    14 0 PRIMB(1) =S(2)*GPRIM(2)
    15 0 PRIMA(1) = FPRIM(2) + (1.-5(2)
    151 ( GGPRIM(2)
    16 0 PRIMB(N) = S(N-1) * GPRIM(N)
    16 1 + FPRIM(N)
    17 O PRIMA(N): (1.-S(N-1))
    17 1 GPRIM(N)
    18 O FABPM = 0.0
    20 0 GAORB = 0.0
    20 0 GAORB = =0.0
    21 0 DO 24 I=I,NN
    22 (FABPM = FAB
    22 1 (l--S(I|))
    23 O FBAPM = FBAPM + (PRIMBII) *
    23 1 S(II)
    240 GAORB = GAORB + (PRIMBII) *
    24 1 (l.-S(I)|
    0000 0 CASE OF UABPM, FBAPM, GAORB
C 0000 O CASE OF UNIT LOAD
C OOOO O CALCULATE MODIFIED ANGULAR
C 0000 O LOAD FUNCTIONS DUE TO UNIT
C 0000 O FORCE APPLIED AT SECTION I 
C 0000 O ALSO MODIFIED DEFLECTION OF
C 0000 0 SECTION I DUE TO UNIT END.
C 0000 O MOMENT
    26 VBARA(2)' = FABPM - PRIMA\1)
    27 O VBARB{2! #,GAORB - PRIMB\1)
    29 0 VBARA(I) = VBARAII-11 -
    29 1 PRIMA(I-1)
    300 VBARB(I) = VBARB(I-1)
    30 1 - PRIMB(I-1)
    31 0 BMA(1) = 0.0
    320 BMB(1)=0.0
    340 BMA(I) = BMA(I-1) + (S(1)
    34 1 S(I-1)) *VBARA(I)
    350 BMB(I) = BMB(I-1)+(S(I)
    351-S(I-1)) VBARB(1)
    36 O PUNCH, (BMA(II, I=ION)
    36 0 PUNCH: (BMA{I\: I=ION)
C OOOO O CASE OF UNIFORM LOAD
C OOOO O CALCULATE MODIFIED ANGULAR
C OOOO O FUNCTIONS DUE TO UNIFORM LOAD
    38 O TABUL = 0.0
    39 0.TEAUL = 0.0
    40 0 DO 42 I=2.K
    410 TABUL = TABUL + (1S(I+1)-
    41:2 S(I-1)),2.0) BMA(I)
    42 0 TBAUL = TBAUL + (1S(I+1) -
    42 1 S(I-1)), 2.1 # BMB(I)
C OOOO O CASE OF DEAD LOAD
C. 0000 O CALCULATE MODIFIEO ANDULAR
C OOOO D FUNCTIONS DUE TO DEAD LOAD
    4 4 0 ~ T A B D L ~ = ~ 0 . 0 ~ 0
    45 O TBADL = 0.0
    46 O DO 4B I=2,K
    470 TABDL = TABDL+(AREAlI)/AREAO*
    47 (S(I)-S(I-1))/2.0 + AREA(1+1)
    4 7 3 \text { BMA(II}
    480 TBADL = TBADL + (AREA(I)/AREAO
    48.1 * (S(I)-S(I-1))/2.0 + AREAI
    48 2 I+1)/ AREAO * (5II+1)-5!I)
    48 3 2.0) * BMB(II
    49 O PUNCH, TABDL, TBADL
    40}50\mathrm{ DO 51 I=2,N
    50 0 DO 51 I=2,N 
    51 52 GO TO 2
    530 END
```

    Figure 3-10
    CHAPTER IV

DEHLECTIONS

## 4-1 Discussion

The total deflection $\Delta_{\mathrm{x}}$ at any section x due to loads can be expressed

$$
\begin{equation*}
\Delta_{x}=B \Delta_{x}+M_{A}\left(\tau_{A B}\right)_{x}+M_{B}\left(\tau_{B A}\right)_{x} \tag{4-1}
\end{equation*}
$$

where: $\quad \triangle_{x}$ - total deflection at ordinate x due to loads
$B \Delta_{x}$ - deflection at ordinate $x$ of simple beam $A B$ due to
$M_{A}$ - bending moment at support $A$ due to loads
$M_{B}$ - bending moment at support $B$ due to loads
$\left(\tau_{A B}\right)_{X}-$ deflection $_{M_{A}}=1$
$\left(\tau_{B A}\right)_{x}-\begin{aligned} & \text { deflection at ordinate } \\ & M_{B}=1\end{aligned}$

A numerical solution for deflection can be obtained only after Phase II has been completed for angular functions. A solution for the final end moments can be obtained by substituting into the three moment equation. The simple beam deflections due to unit moment at $A$ or $B$ are $B M A(I)$ and $B M B(I)$ respectively, from the computer program for angular functions.

The simple beam deflection due to loads is developed as Phase
III. Two types of loading are considered:

1. uniform load
2. dead load

## 4-2 Uniform Load

Typical span $A B$ is loaded by a uniform load (Fig. 4-1), causing deflection of the real structure.

By virtue of the theory of the conjugate beam, the simple beam deflections due to uniform load can be evaluated as the magnitude of the conjugate moment diagram.

A typical elastic weight is from Equation 3-1

$$
\bar{P}_{j}^{U L}=M_{i}^{U L} G_{i j}+M_{j}^{U L} \sum F_{j}+M_{k}^{U L} G_{k j}+\sum \tau_{j}
$$

The bending moment diagram for a simple beam with uniform loading is represented by a second degree parabola. A typical moment may be expressed as

$$
M_{j}^{\mathrm{UL}}=\frac{w L^{2}}{8}\left[4 \frac{x_{j}}{L}\left(1-\frac{x_{j}}{L}\right)\right]
$$

Thus a typical elastic weight is

$$
\begin{align*}
\bar{P}_{j}^{U L} & =\frac{W L^{2}}{8}\left[4 \frac{x_{i}}{L}\left(1-\frac{x_{i}}{L}\right) G_{j}+4 \frac{x_{i}}{L}\left(F_{j}+F_{k}\right)\left(1-\frac{x_{j}}{L}\right)\right. \\
& \left.+4 \frac{x_{k}}{L}\left(1-\frac{x_{k}}{L}\right) G_{k}\right]+\left[\frac{W L j^{3}}{24 E I_{j}}+\frac{W L_{k}^{3}}{24 E I_{k}}\right] \\
\bar{P}_{j}^{U L}= & \frac{W L^{3}}{24 E_{0}} \bar{P}_{j}^{U L^{\prime}} \tag{4-2}
\end{align*}
$$

The conjugate end shear (Fig. 4-2) is

real structure with uniform loading
 shear diagram

deflection
Figure 4-1


$$
\begin{align*}
& \overline{\mathrm{V}}_{\mathrm{AB}}^{\mathrm{UL}}=\sum_{j=1}^{\mathrm{n}} \overline{\mathrm{P}}_{j}^{\mathrm{UL}} \frac{x_{j}^{\prime}}{\mathrm{L}} \\
& \overline{\mathrm{~V}}_{\mathrm{AB}}^{\mathrm{UL}}=\frac{\mathrm{wL}^{3}}{24 \mathrm{EI}} \sum_{j=1}^{n} \frac{x_{j}^{\prime}}{\mathrm{L}} \overline{\mathrm{P}}_{j}^{\mathrm{UL}}{ }^{\prime} \\
& \overline{\mathrm{V}}_{\mathrm{AB}}^{\mathrm{UL}}=\frac{\mathrm{wL}^{3}}{24 E I_{0}} \overline{\mathrm{~V}}_{\mathrm{AB}}^{\mathrm{UL}} \tag{4-3}
\end{align*}
$$

and the conjugate shear at any section is

$$
\begin{align*}
& \bar{V}_{j}^{U L}={\overline{V_{i}}}_{U L}^{U L} \bar{P}_{i}^{U L} \\
& \bar{V}_{j}^{U L}=\frac{W L^{3}}{24 E I_{0}^{U}} \bar{V}_{j}^{U^{\prime}} \tag{4-4}
\end{align*}
$$

The conjugate moment at any section is then the simple beam deflection due to uniform load.

$$
\begin{align*}
& B \Delta{ }_{j}^{U L}=\bar{M}_{j}^{U L}=\bar{M}_{i}^{U L}+\bar{V}_{j}^{-U L} L_{j} \\
& B \Delta \frac{U L}{j}=\frac{W L^{4}}{2 L E I} B \Delta_{j}^{U L^{\prime}} \tag{4-5}
\end{align*}
$$

## 4-3 Dead Load

Typical span $A B$ is loaded by dead load (Fig. 4-3), causing deflection of the real structure. If typical segment ij is considered to be of uniform cross section as in Chapters 2 and 3 , the segment is loaded by uniform load of magnitude $\mathrm{w}_{\mathrm{j}}$ where

$$
w_{j}=e A_{j}
$$

To calculate end shear $V_{A B}^{D L}$, the total uniform load of typical segment $i j$ is considered concentrated at the center of the segment. End shear $V_{A B}^{D L}$ can then be evaluated.

$$
V_{A B}^{D L}=\sum_{j=1}^{n} P_{j}^{D L}\left(\frac{\left(x_{j}^{\prime}\right.}{L}+\frac{\left.L_{j}\right)}{2 L}\right.
$$

$$
V_{A B}^{D I}=L e A_{0} \quad \sum_{j=1}^{n}\left[\frac{A_{j}}{A_{0}} \cdot \frac{L}{L} \cdot \frac{\left(x_{j}^{\prime}\right.}{L}+\frac{L_{j}}{2 L}\right]
$$

$$
\begin{equation*}
v_{A B}^{D L}=L \rho A_{0} V_{A B}^{D L^{\prime}} \tag{4-6}
\end{equation*}
$$

where:

$$
P_{j}^{D L}=w_{j} L_{j}=\rho A_{j} \cdot L_{j}-\text { total weight of segment } i j
$$

$$
\begin{aligned}
& P \text { - unit weight of construction material } \\
& A_{j} \text { - cross sectional area of segment } i j \\
& A_{0} \text { - reference area }
\end{aligned}
$$

The moment at any ordinate $j$ is equal in magnitude to the area of the shear diagram to that point. Although segment if is loaded by uniform load $w_{j}$, the shear at ordinate $j$ is equal to the shear at section $i$ minus the total load on section $i j, P_{j}^{D L}$,

$$
\begin{align*}
& v_{j}^{D L}=V_{i}^{D L}-P_{j}^{D L} \\
& V_{j}^{D L}=L e A_{o} V_{A B}^{D L} \tag{4-7}
\end{align*}
$$

And the moment at any ordinate $j$ is:

$$
M_{j}^{D L}=M_{i}+\frac{L_{i}}{2}\left(V_{i}^{D L}+V_{j}^{D L}\right)
$$



$$
\begin{equation*}
\bar{M}_{j}^{D L}=L^{2} \rho A_{0} \bar{M}_{j}^{D L} \tag{4-8}
\end{equation*}
$$

A typical elastic weight due to dead load (Fig. 4-4) is then from Equation 3-1

$$
\begin{align*}
\bar{P}_{j}^{D L} & =\frac{L}{E I}{ }_{0} \cdot\left[M_{i}^{D L} G_{i}^{\prime}+M_{j}^{D L}\left(F_{j}^{\prime}+F_{k}^{\prime}\right)+M_{k}^{D L} G_{k}^{\prime}\right]+\Sigma \tau_{j}^{D L} \\
\bar{P}_{j}^{D L} & =\frac{e A_{0} L^{3}}{24 E I_{0}} \bar{P}_{j}^{D L^{\prime}} \tag{4-9}
\end{align*}
$$

where:

$$
\begin{aligned}
\Sigma \tau_{j}^{D L} & =\tau_{j}^{D L}+\tau_{k}^{D L} \\
& =\frac{A_{j} \rho L_{j}^{3}}{24 E I_{j}}+\frac{A_{k} \rho L_{k}^{3}}{24 E I_{k}} \\
& =\frac{\rho A_{0} L^{3}}{24 E I_{0}}\left[\frac{A_{i}}{A_{0}}\left(\frac{L}{L}\right)^{3} \frac{I_{0}}{I_{j}}+\frac{A_{k}}{A_{0}}\left(\frac{L_{k}}{L}\right)^{3} \frac{I_{0}}{I_{k}}\right]
\end{aligned}
$$

The conjugate end shear (Fig. 4-4) is

$$
\begin{align*}
& \overline{\mathrm{V}}_{\mathrm{AB}}^{\mathrm{DL}}=\sum_{j=1}^{n} \bar{P}_{j}^{\mathrm{DL}} \frac{x_{j}^{\prime}}{\mathrm{L}} \\
& \overline{\mathrm{~V}}_{\mathrm{AB}}^{\mathrm{DL}}=\frac{\rho_{A_{0}} L^{3}}{24 \mathrm{EI}} \sum_{j=1}^{n} \bar{P}_{j}^{D L^{\prime}} \frac{x_{j}^{\prime}}{\mathrm{L}} \\
& \overline{\mathrm{~V}}_{\mathrm{AB}}^{\mathrm{DL}}=\frac{\rho A_{0} \mathrm{~L}^{3}}{24 \mathrm{EI}_{0}} \overline{\mathrm{~V}}_{\mathrm{AB}} \tag{4-10}
\end{align*}
$$

and the conjugate shear at any section is

$$
\begin{align*}
\bar{V}_{j}^{D L} & =\bar{V}_{i}^{D L}-\bar{P}_{j}^{D L} \\
\bar{V}_{j}^{D L} & =\frac{\rho_{0} L^{3}}{2 L E I_{0}} \bar{V}_{j}^{D L^{\prime}} \tag{4-11}
\end{align*}
$$

The conjugate moment at any section is then the simple beam deflection due to uniform load.

$$
\begin{align*}
& B \Delta{ }_{j}^{D L}=\bar{M}_{j}^{D L}=\bar{M}_{i}^{D L}-\bar{\nabla}_{j}^{D L} L_{j} \\
& B \triangle \frac{D L}{j}=\frac{\rho A_{0} L^{4}}{24 E I_{0}} B \triangle L_{j}^{\prime} \tag{4-12}
\end{align*}
$$

## 4-4 Computer Program

## 4-4-1 Results Of Program

The program presented (Fig's. 4-5, 6) is written for the IBM 650 electronic computer using FORTRAN language and floating point arithmetic. The program yields a solution for:

BMUL(I) - deflection coefficient at ordinate $i$ of simple span $A B$ due to uniform loading
$\mathrm{BMDL}(\mathrm{I})$ - defilection coefficient at ordinate $i$ of simple span $A B$ due to dead loading

4-4-2 Data Required
The input data necessary for computing these deflections are $N$ - total number of segments $S(I)$ - distance from left end of member $A B$ to the cut-off
point at $i$ divided by the span length $L-\frac{x_{i}}{L}$
EYE(I) - moment of inertia of segment i
AREA(I) - cross sectional area of segment $i$
FPRIM(I) - angular flexibility coefficient of segment i
GPRIM(I) - angular carry-over flexibility coefficient of segment i The quantities $\operatorname{EYE}(I)$ and $\operatorname{AREA}(I)$ are answers from Phase I (Sectional Properties). The quantities $\operatorname{FPRIM}(I)$ and $\operatorname{GPRIM}(I)$ are answers from Phase II (Angular Functions). The program has been written so as to take these answer cards directly as data for Phase III (Deflections).

## 4-4-3 Auxiliary Quantities

Other quantities integral in the program include:
PBRUL(I) - elastic weight coefficient due to uniform loading
VABPM - end shear coefficient at A due to dead load
VPM(I) - shear coefficient at section i due to dead load
BMPM(I) - moment coefficient at section i due to dead load
PBRDL(I) - elastic weight coefficient at section i due to dead load

VABDL(I) - conjugate shear coefficient at $A$ due to dead load
VABUL(I) - conjugate shear coefficient at A due to uniform load

```
VDL(I) - conjugate shear coefficient at section i due to
            dead load
VUL(I) - conjugate shear coefficient at section i due to
            uniform load
    EYEO - reference moment of inertia (taken as EYE(2) in
        computer program)
AREAO - reference area (taken as AREA(2) in computer program)
```



Figure 4-5
Flow Chart Phase III-Deflections

```
C 0000 O PHASE 3
C 0000 O SIMPLE BEAM DEFLECTION
OOOO Ó. FOR DEAD LOAD AND UNIFORM LOAD
C 0000 O ELASIIC WEIGHT DUE TO UNIFORM
C 0000 O LOAD
        1 O DIMENSION.S(25), EYE(25),
        1 FPRIM(25), GPRIM(25),
        2 AREA(25). PBRUL(25)
        2 O DIMENSION VPM(25): BMPM(25).
        2.1 PBRDL(25). VDL(25), VUL(25):
        2 BMDL(25): BMUL(25)
        C READ, N
        O READ, (SII): I=1,N)
        50 DO 6 I=2,N
        O READ, EYEIII, AREA(I)
        7 OO. }8\mathrm{ I E2;N
        C READ, FPRIM(II: GPRIMII)
            EYEO = EYE(2)
            AREAO =AREA(2)
            AREAO =
        O 00 11 I=2,K
    &2,
    10 1 (1.-5(I-1)) * GPRIM(I) + S(I)
    102 * {1.-S(1)] * (FPRIM(I)
    10 3 FPRIM(I+1)) + SII+1)*
    104 (1.-5(I+1))* *GPRIM(I+1))
    1q O PBRUL(I) = PBRUL(1) + (EYEO/
    11 1 EYE(I)J* (ISII) - SII-1)I**3*
    12)+(EYEO/EYE(1+1))*11
    11 3 S(I+1)-5(I)!**3.)
    120 PBRUL(1) = (4*/3*)*SI2)*
    12 11.-S(2))*GPRIM(2) + (EYEO/
    12 2 EYE(2))* (S(2)**3.)
    13 0 PBRUL(N) = (4./3.)* S(N-1)**
    31(1,-S(N-1))* GPRIM(N)+ (EYEO/
    13 2 EYE(N)) * ((SIN) - S(N-1))
    13 3**3.1
C 0000 O ELASTIC WEIGHT DUE TO DEAO
C 0000 O LOAD
    14}100\mathrm{ VABPM = 0.0
    16 0 VABPM = VABPM + AREAIII/AREAO*
    16 1 (S(I)-S(I-I)) (11.-S(I)).t
```



```
    17 O VPM(1) = VABPM
    180 00 19 I=20N
    190 VPM(I) = VPM(I-1) - AREA(I) /
    19.1 AREAO * {S(II-S(I-1))
    200 BMPM(1) = 0.0
    21 00 BMPM(I)}=2, =BMP
    220 BMPM(I)=BMPM(I-I) + -5*
    221 (S(I)-S(I-I))* (VPMII-I)+
    222 VPM(11)
    23 0 DO 25 I=2,K _ PREA(I)/AREAO #(IS
    24 0 PBRDL(I)= AREA(I)/AREAO #(IS
    2 (I)) +AREA(I+1)/AREAO *(\S(I+
    2 (I)| +AREA(I+1)/AREAO (\S(I+
    3 1)-S(I))**3.)*(EYEO/EYE(I+1))
    0 PBRDL(1)= PBRDL(I) + 24.*
    25 1 (BMPM(I-1) #GPRIM(I-1)
    25 2 BMPMII) *(FPRIMII) +
    3 FPRIM(I+1)) + BMPM(I+1)*GPRIM(
    I+1!)
    PBRDL(1) = 24.* BMPM(2) *
    GPRIM(2)+AREA(2)/AREAO * (St2)
    6 2**.) * (EYEO/EYE(2))
    270 PBRDL(N) = 24.** BMPM(N-1).*
    27 1 GPRIM(N-1) + AREA(N) , AREAO *
    272 (IS(N)-S(N-1))**3.)* (EYEO/
    27 3 EYE(N))
    8 O VABDL =0.0
    2 9 0 ~ V A B U L ~ = ~ 0 . 0 ~
    30 0 DO 32 I=1,N
    310 VABDL = VABOL+(l.-SII|)
    31 1 PBRDLil)
    32 O VABUL = VABUL. + (1.-5{I)|*
    32 1 PBRUL!I)
    330 VOL(1) = VABDL
    340 VUL(1) = VABUL
    350 DO 37 I= 2,N
    360 VDL(I) = VDL(1-1) - PBRDL(I)
    360}370\mathrm{ VOL(I) = VOL(1-1)= VUL(1-1)-PBRDL(i)
    0 BMOL(1) = 0.0
    390 BMUL(1) = 0.0
    40 00 42 I=2,N
    410 BMDL{I} = BMDL(!-1) + {S(I) -
    4 1 1 5 ( I - 1 ) ) * V D L ( I )
    20 BMUL(I) = BMUL(I-1) + {S\I) -
    4 2 ~ 1 ~ S ( I - 1 ) ) ~ v u l i l )
    430 PUNCH, (BMDL(I), I=2,N
    43 0 PUNCH, (BMDL(I), I=2,N )
    440 PUNCH, (BMUL(1), I=2,N)
    45 O GO TO 3
    46 O END
```

        Figure 4-6
    CHAPTER V

APPLICATION

## 5-1 Introduction

One span from the five span continuous beam Benton Street Bridge in Iowa City, Iowa (1,5) is considered (Fig. 5-1, 2). The bridge, designed by Ned L. Ashton, is an all welded steel deck girder highway bridge completed in July, 1949.

Sectional properties, angular function coefficients, and deflection coefficients are computed using the FORTRAN programs from Chapters 2, 3, 4. Available known results check favorably. Coefficients must of course be multiplied by the corresponding constants to obtain final angular functions or deflections.

## 5-2 Cross Sectional Properties

Data is presented in inches or dimension-less quantities and are of the form:

$$
\begin{aligned}
& N \\
& M(I) \\
& B(I, J), H(I, J), \quad J=1, M(I) \quad-I=2,8
\end{aligned}
$$



| $8+$ |  |
| ---: | ---: |
| $3+$ |  |
| $1800000052+$ | $1250000051+$ |
| $5000000050+$ | $4550000052+$ |
| $1800000052+$ | $1250000051+$ |
| $1700000052+$ | $3593750050+$ |
| $1800000052+$ | $1250000051+$ |
| $5000000050+$ | $4550000052+$ |
| $180000052+$ | $1250000051+$ |
| $1700000052+$ | $3593750050+$ |
| $5+$ |  |
| $1700000052+$ | $5000000050+$ |
| $1800000052+$ | $1250000051+$ |
| $5000000050+$ | $4550000052+$ |
| $1800000052+$ | $1250000051+$ |
| $1700000052+$ | $5000000050+$ |
| $1700000052+$ | $50000000050+$ |
| $1800000052+$ | $1250000051+$ |
| $5000000050+$ | $4550000052+$ |
| $180000052+$ | $1250000051+$ |
| $170000052+$ | $5000000050+$ |
| $3++$ |  |
| $1800000052+$ | $1250000051+$ |
| $5000000050+$ | $4550000052+$ |
| $1800000052+$ | $1250000051+$ |
| $3+$ |  |
| $1800000052+$ | $2000000051+$ |
| $5000000050+$ | $4400000052+$ |
| $1800000052+$ | $2000000051+$ |
| $1700000052+$ | $7500000050+$ |
| $1800000052+$ | $2000000051+$ |
| $500000050+$ | $4400000052+$ |
| $180000052+$ | $2000000051+$ |
| $170000052+$ | $7500000050+$ |
|  |  |

Computed answers are of the form:


```
2851832055+ 6775000052+
4800000052+ 1188263354+ 1188263354+
3566224355+ 7996875052+
4871875052+ 1464004854+ 1464004954+
3851574855+ 8475000052+
4900000052+ 1572071354+ 1572071354+
3851574855+ 8475000052+
4900000052+ 1572071354+ 1572071354+
2851832055+ 6775000052+
4800000052+ 1188263354+ 1188263354+
4166136955+ 9400000052+
4800000052+ 1735890454+ 1735890454+
5681317755+ 1195000053+
4950000052+ 2295482054+ 2295481854+
```


## 5-3 Angular Functions

Sectional properties, moment of inertia and area, are read as data directly from Phase I (Cross-Sectional Properties). The data required is of the form:

$$
\begin{gathered}
N \\
S(I), I=2,8 \\
\operatorname{EYE}(I), \operatorname{AREA}(I), I=2,8
\end{gathered}
$$

8+
$2051282050+2307692350+4358974350+6410256450+8205128250+$ 9166666650+ 1000000051+ 2851832055+ 6775000052+ 3566224355+7996875052+ 3851574855+ 8475000052+ 3851574855+ 8475000052+ 2851832055+6775000052+ 4166136955+9400000052+ 5681317755+ 1195000053+

Computed answers are of the form:
FAB, FBA, GAORB
$\operatorname{BMA}(I), I=2,8$
$\operatorname{BMB}(I), I=2,8$
TABUL, TBAIU
TABDL, TBADL
$\operatorname{FPRIM}(I), \operatorname{GPRIM}(I), I=2,8$

```
2941686450+ 2503127450+ 1360281550+
    4074204449+ 4335788049+ 5173109449+ 4252934349+ 2383419349+
1128728049+ 2100000043+
    3906222847+ 4333126947+ 6991551447+ 7645451547+ 5399194547+
2828764747+ 2804428041+
3279186749+ 3287989649+
2633633251+ 2654712951+
6837606649+ 3418803347+
6834857948+ 3417429048+
5062787849+2531393949+
5062788149+2531394149+
5982906049+2991453049+
2193995749+ 1096997949+
1394351249+6971756048+
```


## 5-4 Deflections

Answer cards FPRIM(I) and GPRIM(I) from Phase II (Angular Functions) are added to the data cards for Phase II to form data cards for Phase III (Deflections).

$$
\begin{gathered}
N \\
S(I), I=2,8 \\
\operatorname{EYE}(I), \operatorname{AREA}(I), \quad I=2,8 \\
\operatorname{FPRIM}(I), \operatorname{GPRIM}(I), \quad I=2,8
\end{gathered}
$$

```
    8+
        2051282050+2307692350+ 4358974350+6410256450+ 8205128250+
9166666650+ 1000000051+
2851832055+ 6775000052+
3566224355+7996875052+
3851574855+ 8475000052+
3851574855+ 8475000052+
2851832055+67775000052+
4166136955+9400000052+
5681317755+ 1195000053+
6837606649+ 3418803349+
6834857948+ 3417429048+
5062787849+2531393949+
5062788149+2531394149+
5982906049+2991453049+
2193995749+1096997949+
1394351249+6971756048+
```

Computed answers are of the form:
$\operatorname{BMDL}(I), I=2,8$
$\operatorname{BMUL}(I), I=2,8$
$1859873350+2043632950+2857046250+2718139550+1649428250+8127633949+$ 8560885643+
$2145835749+2347967349+3368698449+3164053249+1903821649+9138638048+$ 1330000043+

## CHAPTER VI

SUMMARY AND CONCLUSION

## 6-1 Summary

In this thesis three separate programs for the IBM 650 electronic computer have been developed in FORTRAN language utilizing floating point arithmetic. The first program gives a solution for the sectional properties of a typical span from a continuous variable cross section structure. An upper limit of twenty five segments with nine plates per segment has been taken. The IBM 650 could handle a greater number of segments or plates if needed and correctly dimensioned. The second program gives the solution for angular functions for this span. The third program gives a solution for the deflections of this span due to dead and uniform loading.

## 6-2 Conclusion

The analysis of a continuous structure of variable cross section for cross sectional properties, angular functions, and deflections proves to be quite adaptable to solution by electronic computer. Once FORTRAN statements have been processed and an object program obtained, very little machine time is required for a typical solution.

The derivation and application is best suited for a plate girder type structure where the cross section is uniform between cut-off points However, when the cross section between cut-off
points is not uniform, basic assumptions can be made for sectional dimensions, yielding very good results.

An unfortunate characteristic of obtaining deflections is that only those deflections where elastic weights are applied may be calculated. However, when additional considerations of deflection are desirable, additional elastic weights may be considered at these locations.

It is felt by the author that this work can be extended in subsequent research to include the evaluation of influence line ordinates and the development of a design procedure using the electronic computer. It is expected that this research would open the way for the development of an optimum design approach.

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