A COMPUTER STUDY OF THE PYROLYSIS

OF POROUS SOLIDS

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PREFACE

This work represents the initial effort by Oklahoma State University's Engineering Research to provide basic research into pyrolysis and combustion phenomenon in support of its Eglin Project. Therefore, I would like to thank Professor Clarence Johnson and Dr. Vincent Haneman in particular, and Engineering Research in general, for supporting and funding this project.

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NOMENCLATURE

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Symbol	Description
A _i	time-dependent, polynomial coefficient i, for the tem- perature approximation
a i	frequency factor for reaction i, sec ⁻¹
В	x-temperature derivative at heated surface
b i	mole fraction for species i in reaction expression, moles of species i/mole of reactant
C -	constant pressure specific heat, cal/gm-°K
c ₁	mass fraction of solid converted in reaction 1, gm Solid $_2$ /gm Solid $_1$
c ₂	mass fraction of solid converted in reaction 3, gm Solid ₆ /gm Solid ₂
E [*] i	activation energy for reaction i, cal/mole
F	frequency factor parameter for reaction i (see TableVI)
fm	weighting function m
G	heat flux parameter (see Table VI)
G	Gas species i
Hi	non-dimensional heat generation term i
H 1,n	time-dependent polynomial coefficient n for heat gener- ation approximation term i
h	heat transfer coefficient, cal/cm ² -sec-°K
J _{i,n}	time-dependent polynomial coefficient n for approxi- mation term i

Note: For symbols having both a dimensional and non-dimensional representation, an asterisk (*) is used to denote the dimensional value. Table IV defines the non-dimensional quantity.

Symbol .	Description
K	effective thermal conductivity based on constant unit area
K	thermal conductivity of solid based on solid area
k i	rate constant for reaction i, sec^{-1}
k _i	exponential part of the rate constant
k i,j	exponential part of the rate constant at the approxi- mating strip j
L	slab thickness, between insulated and heated surfaces, cm
M _i	molecular weight of species i
m'i	rate of generation of species i per unit volume from all reactions
m'i,j	rate of generation of species i per unit volume from reaction j
N	number of approximating strips
Ne	number of ordinary differential equations, $4N + 2$
Nu	Nusselt number (see Table VI)
Q _i	heat of reaction i, cal/gm of reactant
¢;	rate of generation of heat in reaction i per unit volume
Ç,	rate of generation of heat from all reactions per unit volume
q	heat flux, cal/cm ² -sec
R°	universal gas constant, 1.986 cal/mole-°K
R	heat of reaction i parameter (see Table VI)
Si	Solid species i
T*	temperature, °K
T^{*}_{∞}	convective heat source temperature
t*	time, sec
	xíií-
	مىلە مەرەرى

Symbol	Description
`* ` w*	mass flux, gm/cm ² -sec
x*	slab distance from insulated surface, cm
ρ*	effective density of total solid, mass of solid/constant unit volume
ρ *	actual density of total solid, mass of solid/volume of solid
Subscripts	Description
0	initial value, time = 0
1,2,,6	species number
R R	Radiant heating boundary condition
C	Convective heating boundary condition

CHAPTER I

INTRODUCTION

When a porous solid such as wood, cellulose or a polymer foam is heated, its temperature rises at a rate dependent on the rate of surface heating and upon the rate of heat conduction into the solid. The solid continues to absorb heat until the temperature is high enough for chemical reactions to be important. These reactions cause changes in local temperature and solid composition thus affecting future reactions. If it were possible to specify all reactions and their rates, one could, in principle, predict the behavior of the solid, given the boundary conditions and physical properties.

For many years, experimental studies have been made of thermal decomposition of porous solids. Industries have been concerned with improving fuels such as coke and charcoal. The Forest Service has undertaken extensive investigations to understand the spread of forest fires and the effectivness of flame retardants. Insurance companies and agencies concerned with Building Codes are interested in fire spread in buildings, structural damage to supporting members and effects of different methods of fireproofing. Problems of self-heating and ignition also arise when materials are stored for long periods of time in ships and warehouses. Airlines are concerned with fire spread in the interiors of planes when cloth or plastic materials ignite. The Armed Forces study effects of flame weapons on combustible targets such as buildings,

wooden ships and fuel soaked earth. The study of nuclear weapon effects presents problems involving ignition and sustained combustion. Recently, the aerospace industry has been interested in heating effects on solids because of the use of char-forming ablative heat shields on re-entry spacecraft. Another area of interest is the study of effects of in-depth reactions on solid propellant combustion. All industries are involved with the use of polymer foams. These foams are used for electrical, thermal and acoustical insulation as well as lightweight structural applications. Finally, cellulose products used in the textile and paper industries are subject to pyrolysis reactions. These few examples demonstrate the usefulness of improving the mathematical analysis of pyrolysis of porous solids.

There are two main obstacles in performing an analysis; extreme complexity in the physical processes and mathematical difficulties in solving the resulting equations. In the present field, more is known about the qualitative physical events than has been previously incorporated in the mathematical solution. However, large digital computers have enabled more complex problems to be solved and, therefore, more physically realistic models to be studied. Thus, the purpose of the present work is to incorporate more of the known physical events into the mathematical analysis of porous solid pyrolysis.

This report is composed of four major sections. The first section presents background information as to the experimental results that indicate the important physical events. Also included is a discussion of previous mathematical analyses. The second section outlines the development of the mathematical equations and boundary conditions. The third section describes the method used in solving the

equations and the fourth section presents the results and interpretations of the computer solutions.

Background

Wood has been in use since the beginning of time and is still the widest used porous solid today. As a result, a great deal of experimental work has been done on wood and its components.

The three major components of wood are hemicellulose, cellulose, and lignin with approximately half being cellulose. The thermal decomposition rate of wood closely follows that of cellulose whereas hemicellulose decomposes faster, and lignin, slower. Thus, many experiments are done with cellulose to better understand the pyrolysis of wood and other materials. Also, cellulose can be made into homogeneous samples having a wide choice of closely controlled densities and physical constants.

Browne (5)¹ reviews the theories of the combustion of wood up to 1963 while MacKay (13) gives a review of information on the thermal degradation of cellulose up to 1967. Shafizadeh (17) presented, in 1968, a complete review of the experimental information on the pyrolysis and combustion of cellulosic materials and includes effects of flame-retarding chemicals.

In one type of experiment, a slab of cellulose is placed into a constant temperature furnace that is being flushed with an inert gas. Usually, the temperature at certain points is recorded as well as the

¹Throughout the report, numbers in parenthesis after a proper name refer to an entry in the Bibliography.

weight of the slab. As heat enters the cellulose, its temperature rises and causes pyrolysis reactions which produce a different solid and evolve gases. Since some solid has been changed to a gas the solid becomes more porous. The reactions proceed into the solid while the gases flow through the pores and out the surface. Higher temperatures increase the reaction rate, producing more gases and increasing the porosity. Thus, at a given instant, the solid is in various stages of decomposition ranging from the virgin material deep within the solid, through the more porous reacting solid, to the inert highly porous hot char next to the surface. Of course, continued heating to higher temperatures would eventually decompose the char. However, in most practical situations one is concerned with the events from the initial heating to the production of char.

The gases evolved from the reactions are very important in ignition and combustion studies since these gases may ignite and burn. The first gases that are formed are primarily non-combustible water vapor and carbon dioxide. Next, more combustible carbon monoxide, hydrogen and the highly inflammable levoglucosan, sometimes called tar, are evolved. Also produced are methane, formaldehyde, methanol and acids. Efforts in flame-proofing materials are directed toward producing less tar and more non-combustible gases. Thus, before a complete understanding of ignition and combustion stages can be obtained, a thorough investigation of the initial pyrolysis events must be made.

The first comprehensive analytical model of initial pyrolysis was formulated by Bamford, et al (1), in conjuction with their experiments on wood. Slabs of wood of various thicknesses were heated either on

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both sides by flames from a burner or on one side by a radiant heater. Measurements were made of the central slab temperature, depth of char, and time required to achieve sustained burning for the flame heated slabs and piloted ignition² for the radiantly heated slabs.

In the analysis, the slabs were considered to behave as a onedimensional solid heated by a surface-temperature-dependent convective heat flux. The unsteady heat conduction equation included a term for heat generation within the slab. Heat generation was considered to be from a single exothermic reaction and was expressed as a constant heat of reaction times a reaction rate. The reaction rate was expressed as a first-order, Arrhenius-type reaction.³ In addition to the heat of reaction, Bamford assumed constant thermal conductivity, specific heat, surface heat transfer coefficient, activation energy and pre-exponential factor. The latter two constants were chosen by matching the calculated central slab temperature with the experimentally measured one. The temperature was found from the energy equation by a finite-difference method and the mass loss was determined by a finite-difference approximation to the species conservation equation. The calculations for the central temperature, based on a single overall reaction, did not agree with measured values when different slab thicknesses were used. This indicates that more than one reaction needs to be considered.

Weatherford (21) used the same constants and equations as Bamford

²Piloted ignition occurs when a small flame placed near a heated surface acts as an ignition source for the evolving gases.

³An Arrhenius-type reaction is one that expresses the reaction rate constant as $k = a \exp(-E*/R^{T*})$. See Nomenclature for the definition of the symbols.

and calculated the surface temperature and vapor-generation rate. Source temperatures considered for the convective boundary condition varied between 667° K and 767° K. Calculations were continued until the vapor generation rate reached 10^{-4} gm/cm²-sec, became constant, or reached a maximum. He used a finite-difference technique and showed that undulations which Bamford found in the gas-generation rate curves were caused by too large a step size in the finitedifference approximations. Weatherford proposed that a criterion for wood to sustain stable burning would be the departure of its surface temperature or vapor-generation rate from the solution for the infinite thickness case.

Experiments were performed by Weatherford and Sheppard (22) on convectively heated slabs to determine piloted ignition times. Various slab thicknesses, bulk densities, and heat source temperatures were used. The data showed that small changes in density or slab thickness exerted significant influence on the ignition time. For the range of variables considered, the surface temperature calculated with the heat generation term included was within 10% of the solution for an inert slab.

Lawson and Simms (11) considered piloted and spontaneous ignition⁴ of wood slabs heated on one side by radiation. Times to ignition were recorded for various radiation intensities and different species of wood. By extrapolating graphs of radiation intensity vs. ignition-time out to infinite time, a minimum intensity necessary for ignition was determined for slabs thick enough to be considered

⁴Spontaneous ignition relies on the conditions of the heated material for ignition.

a semi-infinite solid. For most species, a constant minimum intensity was found for each of the two types of ignition.

Martin (14) heated thin cellulose slabs by radiation and measured (not all simultaneously) the spontaneous ignition time, temperatures within the slab, weight loss and composition of the gaseous products for various radiation intensities. The temperature profiles differed considerably from the theoretical ones for a semi-infinite, inert solid having constant thermal properties. A maximum in the rate of evolution of volatile products occurred near the ignition time but no evidence was found for an ignition criterion based on a threshold rate of volatile evolution. Evidence of competing reactions was seen by observing a different final weight of char for different heating rates. Char formed from the levoglucosan reaction was found to be negligible.

In the preceding ignition work, the two types of boundary conditions considered were a constant radiative heat flux and a convective heat flux. The combination of radiation heating with Newtonian cooling at the surface, considered by Lawson and Simms, can be considered as a psuedo-convective case. The factors which seem to be the most important in ignition are the gas composition and generation rate and surface temperature, all of which depend on the nature of the pyrolysis reactions. These factors were considered in developing the results of this study.

To obtain more information on the kinetics of reactions, techniques were developed by the physical chemists. Some of these techniques are Thermogravimetric analysis (TGA) and Differential Thermal analysis (DTA). A better understanding of the nature of the reactions helps to explain the causes of differing experimental results as well

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as determining the effects of adding fire retardant chemicals.

Broido and Kilzer (4) presented a critique on the knowledge of the mechanism of cellulose pyrolysis to 1961 and indicated that more detail was needed in interpreting the reaction mechanism as well as in experimental studies. They pointed out that previous considerations of the reaction products as only gas, tar, and char failed to explain many of the observed experimental results. The important effects of trace impurities on the internal reactions caused many earlier experimental results to be questioned.

Kilzer and Broido (9) present data from a number of cellulose pyrolysis observations made by TGA, DTA, mass spectrometric thermal analysis (MTA) and gas and paper chromatographic analysis of the prod-The results show that as a cellulose sample is heated, a slightucts. ly endothermic reaction starts at about 220° C causing the pure cellulose to lose water and form a "dehydrocellulose." (This is after vaporization of any extra-cellulosic moisture). Next, a strongly endothermic reaction becomes evident at about 280° C in which the remaining pure cellulose depolymerizes and volatilizes as tar, thus competing with the first reaction for the original cellulose. The third major reaction is strongly exothermic and results in the "dehydrocellulose" decomposing into a number of gaseous products and residual char. Each of the three processes actually involve a number of reactions. They concluded that the pyrolysis of cellulose must be described by at least two competing endothermic reactions and a consecutive exothermic reaction.

Broido (3) presents TGA and DTA results of cellulose containing practically no contamination (<.01%) and containing as much as 1.5%

KHCO₃. The results showed that as little as 0.15% contamination can significantly affect the pyrolysis reactions. The addition of 1.5% KHCO₃ lowered by 80°C the temperature at which significant decomposition began but eliminated the flame-producing reactions in favor of those leading to glowing combustion.

Tang and Neill (18), in their experiments on the effects of salts on wood and its components, supported the reaction stages described by Broido. They showed that the effect of the salts was to lower the overall activation energy, stimulating the dehydration reaction.

Further effects of multiple reactions were seen when Murty and Blackshear (15) showed that the overall activation energy decreased with radius for heated cylinders of cellulose. The frequency factor varied unsystematically while incubation of the interior caused the amount of solid that occurred at a given temperature to decrease toward the center. This also suggests that the reaction rate decreased toward the center.

Tinney (20) heated small wooden dowels in a constant temperature furnace and measured the center temperature and rate of weight loss. During his calculations for temperature and weight loss, he changed the overall activation energy, frequency factor and heat of reaction to a new value in an attempt to simulate the apparent changes during wood pyrolysis. He obtained fair agreement between experimental and calculated values for the surface temperature and weight loss but poor agreement for the center temperature. He concluded that more than one change in the parameters would be necessary to match the center temperatures for exothermic reactions.

Thus, investigations into the reaction mechanisms of cellulose

have shown the existence of and provided information on multiple reactions while pyrolysis experiments have shown the need for consideration of more than one overall reaction.

The same pattern of three major reactions in porous cellulose is found in the study of the pyrolysis of polymer foams. Hilado (8) describes the combustion process of urethane foams which parallels similar events seen in cellulose and other organic polymers. Tilley, et al (19), include DTA and TGA analysis of the pyrolysis of urethane foams and find three major regions of weight loss as a function of temperature. They associate the weight loss regions with three major reactions whose rates were fitted to the Arrhenius equation. Learmonth and Osborn (12) present the results of the pyrolysis of phenolic resins and describe three main types of products; gases, tars, and carbon residue. As with cellulose, the above references indicate that the nature of the products and the rate of decomposition of polymer foams vary with temperature, rate of temperature rise, and endothermic and exothermic reactions. Thus, learning how different factors influence the pyrolysis of cellulose will contribute to the understanding of the pyrolysis of many organic polymers as well as cellulose and wood.

In summary, the mathematical analysis of the pyrolysis of cellulose or wood has been limited to consideration of the unsteady heat conduction equation with heat generation from one exothermic reaction. Experiments have shown that more than one reaction needs to be considered and should include competing and consecutive, endothermic and exothermic reactions. The effect of a density-dependent effective thermal conductivity also needs to be studied. Also, since previous

mathematical studies involved specific ignition events with limited parameter values, a general analysis is needed to establish the relevant parameters and present results of the influence of parameter variations on the pyrolysis events. The development and solution of the equations as given in the remaining sections will be pointed toward these objectives.

CHAPTER II

THE MATHEMATICAL ANALYSIS

Geometry

As shown in Figure 1, the porous solid is considered to be an infinite slab of finite thickness, L. The slab is heated at the front surface and insulated against heat or mass transfer at the back surface. These conditions also apply to one-half of a slab of thickness 2L that is symmetrically heated.

Two types of heating conditions will be considered for depositing energy at the front surface. The first is a time-dependent, radiative heat flux and the second is a convective heat flux.

Energy Equation

Heat is transferred into the slab from the surface by conduction and causes chemical reactions which either absorb or generate heat. Gaseous products that are formed from the reactions flow out of the solid. It is assumed that any heat transferred between the flowing gases and the solid does not significantly change the solid temperature. This is approximately true since the increasing porosity and cracks allow the gas to flow out with little solid area contact and short residence time.

Specific heat and thermal conductivity of the solid material are considered constant for the ranges in temperatures to be encountered.

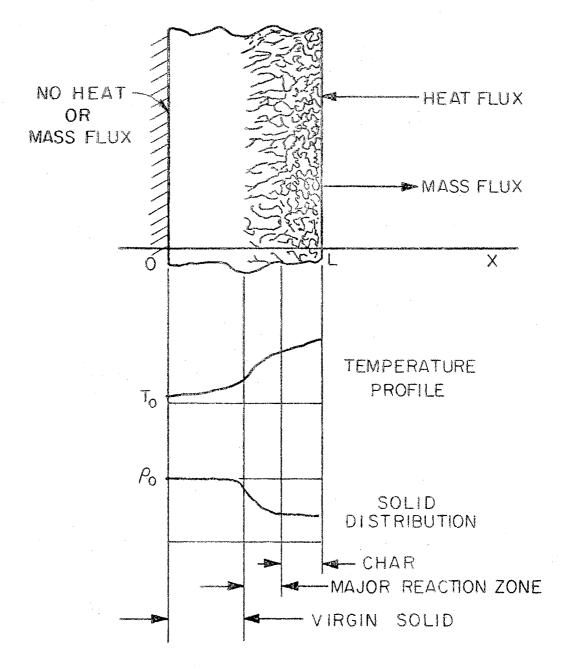


Figure 1. Porous Solid Being Heated

Also, the heats of reaction per mass of reactant are constant. Finally, the dimensions of the slab are assumed constant and a psuedo-density ϕ , based on a constant unit volume, will be used.

With these considerations, the energy equation for the solid can be written by applying the conservation of energy to a differential volume and taking the limit as the thickness goes to zero. The result is valid at every x-position within the solid and is given by

$$\partial^{*}C\frac{\partial T^{*}}{\partial t^{*}} = \frac{\partial}{\partial x^{*}}(K\frac{\partial T^{*}}{\partial x^{*}}) + \dot{Q}^{*}$$
(2.1)

III

II

Term I = Rate of accumulation of thermal energy per unit volume Term II = Net rate of conduction of thermal energy per unit volume Term III = Rate of heat generation by chemical reaction per unit volume

Τ

Effective Thermal Conductivity

The cross section of a porous solid can be divided into two areas; the area occupied by solid material, A solid, and the void area occupied by gas (see Figure 2).

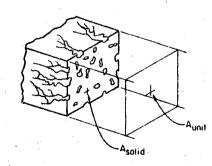


Figure 2. Representation of Unit and Solid Areas

Heat conduction in the slab occurs only through the solid area, A_{solid}. Since the solid area changes, it is more convenient to base the equations on the total unit area, A_{unit}, which is constant because of the assumption that the slab dimensions are constant. Thus, an effective conductivity, K, based on the total unit area is defined by

$$K A_{unit} = \overline{K} A_{solid}$$

where \overline{K} is the thermal conductivity of the solid material and is considered constant. The ratio of the areas can be expressed as a density ratio by considering a unit thickness. Then,

$$\frac{A_{\text{solid}}}{A_{\text{unit}}} = \frac{\rho^*}{\rho^*}$$

where $\overline{\rho}$ * is the density of the solid material (i.e., no porosity) and is considered constant.

With the above equations, an expression for the effective thermal conductivity becomes

$$K = K_0 \frac{\rho^*}{\rho_0^*} \tag{2.2}$$

where K_0 and $\rho_0^{\, {\star}}$ are the effective conductivity and density at time 0.

Substituting Equation (2.2) into term II of Equation (2.1), expanding and dividing by $\rho*C$ gives

$$\frac{\partial \mathbf{T}^{*}}{\partial \mathbf{t}^{*}} = \frac{K_{0}}{\rho_{0}^{*}C} \frac{\partial^{2}\mathbf{T}}{\partial \mathbf{x}^{*2}} + \frac{K_{0}}{\rho_{0}^{*}C\rho^{*}} \frac{\partial \rho^{*}}{\partial \mathbf{x}^{*}} \frac{\partial \mathbf{T}^{*}}{\partial \mathbf{x}^{*}} + \frac{\dot{\mathbf{Q}}^{*}}{\rho^{*}C}$$
(2.3)
I IIa IIb III

Previous investigators did not consider that the changing effective

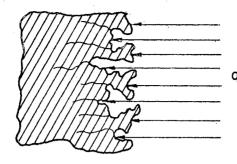
thermal conductivity affected the energy equation. They argued incorrectly that since the thermal diffusivity, $K/\rho *C$, remained constant, then term IIa was the only conduction contribution. This approximation can only be true if term IIb is negligible. From another viewpoint, we might consider that previous workers have unintentionally combined terms IIb and III into a new, equivalent, heat generation term. Term IIb will always be negative or zero and thus introduces an equivalent endothermic heat generation term that will vary throughout the slab as well as time. As a result, heat generation calculations based on an energy equation that considers the effective thermal conductivity to be a constant may calculate values that largely include a distance and time dependency from the conductivity change. This investigation will include calculations to determine how a density-dependent thermal conductivity may affect the results.

Boundary Conditions

Consideration of a surface with changing porosity introduces several possibilities for the surface heating conditions. For the radiant heat flux¹ condition, all the energy strikes the solid either at the surface or at some small depth due to the pores (see Figure 3).

It is assumed that the depth at which the energy is deposited is small enough to consider it all at the surface. Therefore, the amount of radiant energy striking the surface is not dependent on the amount of solid present. The rate at which the energy is conducted into the solid, however, is dependent on the porosity through the effective

 $^{1}\mbox{Heat}$ flux refers to the total energy per constant unit area per unit time.



q* RADIANT HEAT FLUX

Figure 3. Radiant Heat Flux Deposited at the Surface

thermal conductivity. Thus, the first possibility is a time-dependent heat flux at the surface with a density-dependent, effective thermal conductivity. This condition will be called Case 1. For comparison purposes, Case 0 will denote problems with the heat flux treated as in Case 1 but with the effective thermal conductivity considered constant.

Another possibility is when the energy deposited at the surface decreases in proportion to the amount of solid at the surface. This is difficult to justify physically but might approximate a radiative heat flux reduced by opaque gases flowing from the solid or, if energy was lost from the surface by re-radiation and conduction to the gases at an equivalent rate. The resulting condition is the same as Case 0 but with term IIb included in the energy equation (2.3). Thus, the contribution of the change in the effective thermal conductivity to the energy equation can be found. This condition will be called Case 2.

For the convective heat flux condition, three cases also are developed. Case 0 considers no change in the effective thermal conductivity and a constant convective heat transfer coefficient. Case 1 considers a density-dependent, effective thermal conductivity and a constant convective heat transfer coefficient. This is equivalent to saying that the convective fluid at the surface is always in contact with the solid as the surface becomes more porous. Case 2 considers that the convective heat transfer coefficient and the effective thermal conductivity decrease as the solid at the surface decreases. This would mean that the convective gases were in contact with the solid that was present only at the surface (see Figure 4).

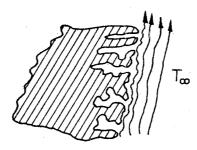


Figure 4. Convective Heat Flux at the Surface

Cases 1 and 2 represent an upper and lower bound on the physical events for the convective case with the intermediate Case 0 probably being a good assumption.

At the instant heat is applied, the slab, $solid_1$, is at a constant temperature and density. With the insulated boundary at $x^* = 0$ and the heated surface at $x^* = L$, the boundary conditions for radiant and convective heating become

For time = 0

$$T^*(x^*, 0) = T_0^*$$
 (2.4a)

$$\rho^*(\mathbf{x}^*, 0) = \rho_1^*(\mathbf{x}^*, 0) \tag{2.4b}$$

$$\rho_i^*(\mathbf{x}^*, 0) = 0, \quad i = 2, 3, \ldots, 6 \quad (2.4c)$$

At $x^* = 0$

Case

$$\frac{\partial T^*}{\partial x^*} = 0 \tag{2.4d}$$

$$\frac{\partial \rho^*}{\partial x^*} = 0 \tag{2.4e}$$

At x* = L: Type I (Radiative)

Case 0 $K_0 \frac{\partial T^*}{\partial x^*} = q^*(t^*)$ (2.4f)

1
$$K_0 \frac{\rho^*}{\rho_0^*} \frac{\partial T^*}{\partial x^*} = q^*(t^*)$$
 (2.4g)

Case 2
$$K_0 \frac{\partial T^*}{\partial x^*} = q^*(t^*)$$
 (2.4h)²

At x* = L: Type II (Convective)

Case 0
$$K_0 \frac{\partial T^*}{\partial x^*} = h(T^*_{\infty} - T^*)$$
 (2.4i)

Case 1
$$K_0 \frac{\rho^*}{\rho_0^*} \frac{\partial T^*}{\partial x^*} = h(T_\infty^* - T^*)$$
 (2.4j)

Case 2
$$K_0 \frac{\partial T^*}{\partial x^*} = h(T^*_{\infty} - T^*)$$
 (2.4k)²

²Equations (2.4h,k) would normally have the term $\rho*/\rho*$ multiplying the terms on both sides of the equal sign but is shown already canceled.

Equation (2.4) represents the boundary conditions that will be used with the energy and species equations for completely specifying the problem. Equation (2.4e) can be determined by taking the x-derivative of the species equations (see Appendix F).

Chemical Reactions

The reactions that will be considered are shown in Figure 5 with identification of each species for the case of cellulose. Of course, the same reaction scheme or any combination of the reactions may be used for many other materials.

Even though these reactions represent only the overall effects of a number of reactions, they include the major pyrolysis events in most porous solids. Also, three competitive and consecutive reactions are just enough to enable a large number of possible effects to be studied.

Steps leading to expressions for the heat generated by the reactions and the rate of species production begin with writing the reactions on a mole basis:

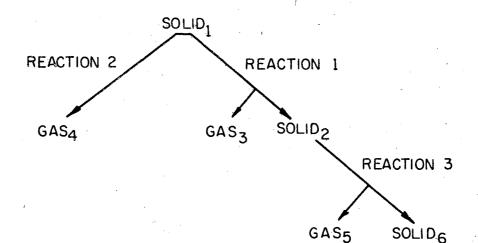
$$s_1 \xrightarrow{k_1} b_2 s_2 + b_3 G_3$$

$$s_1 \xrightarrow{k_2} b_4G_4$$

$$s_2 \xrightarrow{k_3} b_6 s_6 + b_5 g_5$$

where S = Solid, G = Gas, k_i = rate of reaction i, $b_i = \frac{\text{moles of Species i}}{\text{mole of reactant}}$. Let M_i = Molecular weight of Species i. Then, mass is conserved if

$$M_1 = b_2 M_2 + b_3 M_3$$
 for reaction 1



REACTIONS 1 and 2 are COMPETITIVE

REACTIONS 1 and 3 are CONSECUTIVE

For Cellulose:

Solid1 = original cellulose
Solid2 = "dehydrocellulose"
Gas3 = water vapor
Gas4 = "tar" (levoglucosan)
Gas5 = C0, C02, H20, etc.
Solid6 = char

Figure 5. Reactions in the Slab

$$M_{1} = b_{4}M_{4} \qquad \text{for reaction } 2$$

$$M_{2} = b_{6}M_{6} + b_{5}M_{5} \qquad \text{for reaction } 3.$$

$$c_{1} = b_{2}M_{2}/M_{1} = gm \ S_{2}/gm \ S_{1} \qquad 0 \le c_{1} \le 1$$

$$c_{2} = b_{6}M_{6}/M_{2} = gm \ S_{6}/gm \ S_{2} \qquad 0 \le c_{2} \le 1$$

the reactions may then be written on a gram basis noting that $1 = b_4 M_4 / M_1$

Letting

$$S_1 \xrightarrow{k_1} c_1 S_2 + (1 - c_1) G_3 \qquad (2.5a)$$

$$S_1 \xrightarrow{k_2} G_4$$
 (2.5b)

$$s_2 \xrightarrow{k_3} c_2 s_6 + (1 - c_2) G_5$$
 (2.5c)

It is assumed that all reactions are first-order³ homogeneous reactions, i.e., the rate of decrease of reactant is proportional to the amount of reactant present. The proportionality constant is given by the reaction rate constant. It is expected that basic homogeneous reactions will be first order. However, when several reactions are occurring simultaneously, the resulting overall effect may be of zero order or some intermediate value. Shafizadeh (17) discusses experimental results that determine the order of cellulose reactions, most of which indicate the reactions to be first order with some reports of zero order depending on the experiment. Most analyses also consider the reactions to be first order.

 $^3 Order$ refers to the exponent on the mass term in the reaction rate equation, $(d\rho_1^{*})/(dt^{*})$ = $-\rho_1^{*}k_1$,

With the reaction order specified, the rate of species production can be expressed.

Let m' = rate of generation of species i per unit volume as a result of reaction j. Then,

$$\dot{m}_{1,1}^{\prime} = -\rho_{1}^{*}k_{1} ; \qquad \dot{m}_{2,1}^{\prime} = c_{1}\rho_{1}^{*}k_{1} ; \qquad \dot{m}_{3,1}^{\prime} = (1 - c_{1}) \rho_{1}^{*}k_{1}$$
$$\dot{m}_{1,2}^{\prime} = -\rho_{1}^{*}k_{2} ; \qquad \dot{m}_{4,2}^{\prime} = \rho_{1}^{*}k_{2}$$
$$\dot{m}_{2,3}^{\prime} = -\rho_{2}^{*}k_{3} ; \qquad \dot{m}_{6,3}^{\prime} = c_{2}\rho_{2}^{*}k_{3} ; \qquad \dot{m}_{5,3}^{\prime} = (1 - c_{2}) \rho_{2}^{*}k_{3}$$

Now, the total rate of generation of species i is found by summing over all reactions.

$$\dot{m}_{i} = \sum_{j} \dot{m}_{i,j}$$

$$\dot{m}_{1} = -\rho_{1}^{*}k_{1} - \rho_{1}^{*}k_{2}$$
 (2.6a)

$$\dot{m}_2' = c_1 \rho_1^* k_1 - \rho_2^* k_3$$
 (2.6b)

$$\dot{m}_3 = (1 - c_1) \rho_1^* k_1$$
 (2.6c)

$$\dot{m}_{4}' = \rho_{1}^{*}k_{2}$$
 (2.6d)

$$\dot{m}_5 = (1 - c_2) \rho_2^* k_3$$
 (2.6e)

$$\mathbf{m}_{6}^{*} = \mathbf{c}_{2}\rho_{2}^{*}\mathbf{k}_{3}$$
 (2.6f)

For the heat of reaction given as Q_i = heat generated in reaction i per unit mass reacted and for $\dot{Q}_i^{'}$ = rate of generation of heat per unit volume in reaction i, then,

$$\dot{Q}_1 = Q_1 \dot{m}_{1,1}$$
; $\dot{Q}_2 = Q_2 \dot{m}_{1,2}$; $\dot{Q}_3 = Q_3 \dot{m}_{2,3}$.

Summing over all reactions, $\sum_{i} \dot{Q}'_{i}$, gives the total rate of generation of heat per unit volume, \dot{Q}' , or also

$$\dot{Q}' = -Q_1 \rho_1^* k_1 - Q_2 \rho_1^* k_2 - Q_3 \rho_2^* k_3$$
 (2.7)

For the reaction rate constant, k_i , the usual assumption is made that the dependency of the rate of reaction i on the temperature is given by the Arrhenius expression, $k_i = a_i \exp(-E_i^*/R^\circ T^*)$.

Species Equations

The conservation of a solid species within a unit volume requires that the rate of accumulation of a species within the volume equals the rate of production of the species from chemical reactions. Taking the limit as the volume thickness goes to zero gives an equation that is valid at any x-position,

$$\frac{\partial \rho_1^*}{\partial t^*} = \mathbf{m}_1'$$

Since the species production equations are given by Equation (2.6), the rate of change of each solid species at any position within the slab is

$$\frac{\partial \rho_1^*}{\partial t^*} = -\rho_1^* k_1 - \rho_1^* k_2$$
 (2.8a)

$$\frac{\partial \rho_2^*}{\partial t^*} = c_1 \rho_1^* k_1 - \rho_2^* k_3 \qquad (2.8b)$$

$$\frac{\partial \rho_6^*}{\partial t^*} = c_2 \rho_2^* k_3 \tag{2.8c}$$

Non-Dimensional Equations

The equations and boundary conditions are non-dimensionalized in order to determine relevant parameter groupings and minimize the number of parameters which affect the calculations. Reference values used to non-dimensionalize the variables are selected as being a characteristic dimension of the problem. Usually, the reference value will enable the non-dimensional quantity to have an order of magnitude of unity. For the slab, the independent variables, x* and t*, are non-dimensionalized by the slab thickness and the characteristic conduction time, the Fourier number. The dependent variables ρ_1^* are non-dimensionalized by the initial density, ρ_0^* . The temperature is non-dimensionalized by consideration of the boundary conditions. The resultant non-dimensional variables are given in Table I (see Nomenclature for symbol identification).

TABLE I

NON-DIMENSIONAL VARIABLES

Dependent Variables

$$T_{R} = \frac{T^{*} - T_{0}^{*}}{q_{0}^{*}L/K_{0}}$$

$$T_{C} = \frac{T^{*} - T_{0}^{*}}{T_{\infty}^{*} - T_{0}^{*}}$$

$$q = \frac{q^{*}}{q_{0}^{*}}$$

$$\rho_{i} = \frac{\rho_{i}^{*}}{\rho_{0}^{*}}$$

$$w = \frac{w^{*}CL}{K_{0}}$$

Independent Variables

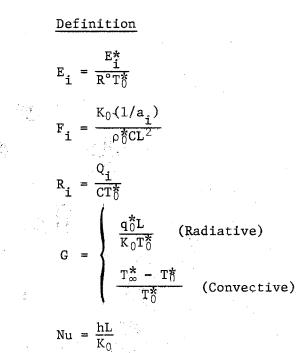
$$x = \frac{x^*}{L}$$

$$t = \frac{K_0 t^*}{\rho_0^* CL^2}$$

When the non-dimensional variables are substituted into the energy equation (2.3), the species equation (2.8), and the boundary conditions (2.4), the reference quantities become grouped into nondimensional parameters. It is the value of the particular combination of reference quantities as given in the new parameters that determine the character of the solutions rather than the values of the reference quantities themselves. The resulting new parameters are given in Table II.

TABLE II

NON-DIMENSIONAL PARAMETERS



Activation Energy Parameter

Description

Frequency Factor Parameter

Heat of Reaction Parameter

Heat Flux Parameter

Nusselt Number

The resulting non-dimensional energy equation is

$$\frac{\partial T}{\partial t} = \frac{\partial^2 T_2}{\partial x^2} + \frac{1}{\rho} \frac{\partial \rho}{\partial x} \frac{\partial T}{\partial x} - \frac{R_1}{F_1 G} \left(\frac{\rho_1}{\rho} \right) \overline{k}_1 - \frac{R_2}{F_2 G} \left(\frac{\rho_1}{\rho} \right) \overline{k}_2 - \frac{R_3}{F_3 G} \left(\frac{\rho_2}{\rho} \right) \overline{k}_3 \quad (2.9)$$

The non-dimensional species equations are

$$\frac{\partial \rho_1}{\partial t} = -\rho_1 \left(\frac{\overline{k}_1}{\overline{F}_1} + \frac{\overline{k}_2}{\overline{F}_2} \right)$$
(2.10a)

$$\frac{\partial \rho_2}{\partial t} = c_1 \rho_1 \frac{\overline{k}_1}{\overline{F}_1} - \rho_2 \frac{\overline{k}_3}{\overline{F}_3}$$
(2.10b)

$$\frac{\partial \rho_6}{\partial t} = c_2 \rho_2 \frac{k_3}{F_3}$$
(2.10c)

where $\overline{k}_{i} = \exp[-E_{i}/(1 + GT)]$. Finally, the non-dimensional boundary conditions are

t = 0:

$$T(x, 0) = 0$$
 (2.11a)

$$\rho(\mathbf{x}, 0) = 1 = \rho_1(\mathbf{x}, 0) \tag{2.11b}$$

$$\rho_i(x, 0) = 0$$
, $i = 2, 3, ..., 6$ (2.11c)

x = 0:

$$\frac{\partial \mathbf{T}}{\partial \mathbf{x}} = 0 \tag{2.11d}$$

$$\frac{\partial \rho}{\partial x} = 0$$
, $i = 1, 2, ..., 6$ (2.11e)

$$x = 1$$
: Type I (Radiative)

Case 0 $\frac{\partial T}{\partial x} = q(t)$ (2.11f)

Case 1
$$\frac{\partial T}{\partial x} = \frac{q(t)}{\rho(1, t)}$$
 (2.11g)

Case 2
$$\frac{\partial T}{\partial x} = q(t)$$
 (2.11h)

x = 1: Type II (Convective)

Case 0
$$\frac{\partial T}{\partial x} = Nu (1 - T)$$
 (2.111)

Case 1
$$\frac{\partial T}{\partial x} = \frac{Nu (1 - T)}{\rho(1, t)}$$
 (2.11j)

Case 2 $\frac{\partial T}{\partial x} = Nu (1 - T)$ (2.11k)

Therefore, the problem to be solved consists of Equations (2.9), (2.10), and (2.11).

CHAPTER III

SOLUTION OF THE EQUATIONS

Background

There are basically two approaches available for the computer solution of non-linear partial differential equations. The first is the finite-difference method where the derivatives are replaced by finite differences. At each step, a system of linear algebraic equations must be solved. Computer programs are usually complex and require large core storage. The second approach is the integral method. Basically, it reduces the partial differential equations to a system of ordinary differential equations by integrating over one variable. The ordinary differential equations can then be solved by any one of a number of highly developed techniques.

There have been a number of different integral approaches used to reduce the partial differential equations. The simplest approach is to divide the region of integration into strips and replace the derivatives across the strips by finite differences. However, a large number of strips are required to obtain good accuracy. Other simple techniques have been discussed by Goodman (7).

Method of Integral Relations

Berlotserkovskii and Chushkin (2) present the essential features of the method of Integral Relations, as generalized by Dorodnitsyn, and

discuss a number of sample problems in the field of Gas Dynamics. This method offers the best accuracy and the fewest number of necessary approximating strips of any of the integral techniques.

The basic approach is to multiply the partial differential equation by a "weighting" function, f(x), and then integrate with respect to its independent variable. Functions within the integrals are then approximated by an interpolation formula. This enables the integrals to be evaluated and results in a system of ordinary differential equations in the remaining independent variable. The final system of equations can then be solved by such standard techniques as Runga-Kutta or an Adams method. The accuracy of the method is improved by increasing the number of strips in the region.

In contrast to a finite difference technique, this method approximates an integral instead of a derivative. Thus, the approximated function is smoother than the integrand and can be represented by a smaller number of interpolation strips. Also, the integral can be continuous even when the integrand is discontinuous. Another consideration is that computer storage requirements are small.

The weighting functions used to multiply the partial differential equations are arbitrarily chosen except for some general restrictions. They must generate as many independent partial differential equations as there are interpolating points and thus must be linearly independent. They are also used to insure convergence of an integrand at any point within the region. Finally, they should enable the approximating functions to be more accurate in regions of greater importance.

Application of the Method of Integral Relations

The energy equation (2.9) along with the boundary conditions (2.11) will be reduced by applying the method of Integral Relations to the x-variable. The Integral method will not need to be applied to the species equations (2.10) since they do not contain derivatives with respect to x.

The first step is to multiply the energy equation (2.9) by a weighting function $f_m(x)$ and then integrate between the slab boundaries. The result is

$$\int_{0}^{1} f_{m}(x) \frac{\partial T}{\partial t} dx = \int_{0}^{1} f_{m}(x) \frac{\partial^{2}T}{\partial x^{2}} dx + \int_{0}^{1} f_{m}(x) \frac{1}{\rho} \frac{\partial \rho}{\partial x} \frac{\partial T}{\partial x} dx$$
$$- \frac{1}{G} \sum_{i=1}^{S} \int_{0}^{1} f_{m}(x) \overline{H}_{i} dx \qquad (3.1)$$

where

$$\overline{H}_{1} = \frac{R_{1}}{F_{1}} \left(\frac{\rho_{1}}{\rho}\right) \overline{k}_{1}$$
(3.2a)

$$\overline{H}_2 = \frac{R_2}{F_2} \left(\frac{\rho_1}{\rho}\right) \overline{k}_2$$
(3.2b)

$$\overline{H}_{3} = \frac{R_{3}}{F_{3}} \left(\frac{\rho_{2}}{\rho}\right) \overline{k}_{3}$$
(3.2c)

Next, the slab is divided into (N - 1) strips by (N - 2) equally spaced planes parallel to the boundaries x = 0 and x = 1. With the boundaries, this gives N points where the equations and boundary conditions will be applied, $(N \ge 2)$. Therefore, there must be N independent equations like equation (3.1). This is obtained by choosing a system of linearly independent weighting functions, $f_m(x)$, with m = 1, 2, ..., N. Since the temperatures increase toward the heated surface, x = 1, this region will have a greater importance in the problem than the back surface, x = 0. Therefore, the weighting functions will be selected to give more "weight" to this region and enable the approximations to be more accurate. The linearly independent functions are chosen as

$$f_{m}(x) = [1, x, x^{2}, ..., x^{(N-1)}]$$

i.e., $f_{m}(x) = x^{(m-1)}$, $(m = 1, 2, ..., N)$ (3.3)

With this set of f_m 's, Equation (3.1) becomes¹

$$\frac{d}{dt} \int_{0}^{1} x^{(m-1)} T dx = \frac{\partial T}{\partial x} \Big|_{x=1}^{2} - (m-1) T \Big|_{x=1}^{2} + \left\{ \begin{array}{c} T \Big|_{x=0} ; m \neq 2 \end{array} \right\} \\ + (m-1)(m-2) \int_{0}^{1} x^{(m-3)} T dx + (\ln \rho \cdot \frac{\partial T}{\partial x})_{x=1} \\ - \int_{0}^{1} x^{(m-1)} \ln \rho \cdot \frac{\partial^{2} T}{\partial x^{2}} dx \\ - (m-1) \int_{0}^{1} \ln \rho \cdot x^{(m-2)} \frac{\partial T}{\partial x} dx \\ - \frac{1}{G} \sum_{i=1}^{3} \int_{0}^{1} x^{(m-1)} \overline{H}_{i} dx , (m = 1, 2, ..., N) \end{array}$$

(3.4)

In the next step, the integrands are approximated by polynomials in x with time-dependent coefficients. The number of terms will equal the number of interpolation points, N, plus any additional terms that can be determined from applying the boundary conditions. Thus, the

¹See Appendix A.

temperature can be approximated with a polynomial having (N + 2) terms since two boundary conditions for the temperature derivative are known. The approximations will then be

$$T(x, t) = \sum_{n=1}^{N+2} A_{(n)}(t) \cdot x^{(n-1)}$$
(3.5a)

$$\ln \rho \cdot \frac{\partial^2 \mathbf{T}}{\partial \mathbf{x}^2} = \sum_{n=1}^{N} J_{1(n)}(t) \cdot \mathbf{x}^{(n-1)}$$
(3.5b)

$$\ln \rho \cdot \frac{\partial \mathbf{T}}{\partial \mathbf{x}} = \sum_{n=1}^{N} J_{2(n)}(t) \cdot \mathbf{x}^{(n-1)}$$
(3.5c)

$$\overline{H}_{i} = \frac{R_{i}}{F_{i}} \sum_{n=1}^{N+1} H_{i(n)}(t) \cdot x^{(n-1)}$$
(3.5d)

The problem now centers around finding the polynomial coefficients, and thus, the temperature as given by Equation (3.5a). Substituting the approximations (3.5) into Equation (3.4) and integrating² gives N equations for the temperature coefficients, $A_{(n)}(t)$:

$$\frac{1}{m} \frac{dA}{dt} (1) + \sum_{n=1}^{N} (\frac{1}{m+n+1}) \frac{dA}{dt} (n+2) = (1 + \ln \rho) \left| B \right|_{x=1}$$

$$- (m-1) \sum_{n=1}^{N} (\frac{n+1}{m+n-1}) A_{(n+2)}$$

$$- \sum_{n=1}^{N} (\frac{1}{m+n-1}) J_{1} (n)$$

$$- (m-1) \sum_{n=1}^{N-1} (\frac{1}{m+n-1}) J_{2} (n+1)$$

$$- \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \sum_{n=1}^{N+1} (\frac{1}{m+n-1}) H_{i}(n)$$

$$(m = 1, 2, ..., N) \quad (3.6)$$

²See Appendix B.

Another equation must be included with Equation (3.6) in order to solve for the (N + 1) unknowns. This equation is formed by using the temperature approximation (3.5a) in one of the boundary conditions (2.11f-k), depending on the Type and Case, and then differentiating with respect to time. The results for Case 1 are

$$\sum_{n=1}^{N+2} (n - 1) \frac{dA(n)}{dt} = (q) \left(\frac{-1}{\rho^2(1, t)}\right) \frac{d\rho(1, t)}{dt} + \frac{1}{\rho} \frac{dq}{dt}, \text{ Type I (3.7a)}$$

$$\sum_{n=1}^{N+2} \left(\frac{N}{\rho} + n - 1 \right) \frac{dA(n)}{dt} = \left[\frac{-N(1 - T)}{\rho^2} \frac{d\rho}{dt} \right]_{x=1}, \text{ Type II}$$
(3.7b)

Equations (3.6) and (3.7) form a set of (N + 1) equations each containing (N + 1) ordinary first-order derivatives of the temperature coefficients on the left side. The derivatives are equated to terms composed only of the temperature coefficients and species densities. Thus, this system of equations can be solved simultaneously to give an explicit equation in canonical form for each of the derivatives of the temperature coefficients.

The time-dependent coefficients in Equations (3.5b, c, d) are determined by evaluating the approximations at each of the N strips, $x = x_i$. For example, Equation (3.5b) would become

$$\sum_{n=1}^{N} J_{1(n)}(t) \cdot x_{j}^{(n-1)} = \ln \rho (x_{j}, t) \cdot \sum_{n=1}^{N+2} (n-1) A_{(n)}(t) \cdot x_{j}^{(n-2)}$$

$$(j = 1, 2, ..., N)$$

A set of algebraic equations are thus formed for each approximation and can be solved simultaneously for the coefficients. The additional term in Equation (3.5d) is found by taking the x-derivatives of the

approximation and the approximated function, Equation (3.2), and applying the boundary conditions (2.11d and e).

The remaining unknowns are the solid species 1, 2, and 6. As stated previously, the equations defining the time derivatives of the species, Equation (2.10), do not contain x-derivatives and thus do not need to be integrated with respect to x. By treating x as a parameter, the species equations are applied at each x_j to obtain a set of N first order ordinary differential equations already in canonical form for each solid species. Thus, at $x = x_j$, j = 1, 2, ..., N

$$\frac{d\rho_{1,j}}{dt} = -\rho_{1,j} \left(\frac{\overline{k}_{1,j}}{F_1} + \frac{\overline{k}_{2,j}}{F_2} \right)$$
(3.8a)

$$\frac{d\rho_{2,j}}{dt} = c_1 \rho_{1,j} \frac{\overline{k_{1,j}}}{F_1} - \rho_{2,j} \frac{\overline{k_{3,j}}}{F_3}$$
(3.8b)

$$\frac{d\rho_{6,j}}{dt} = c_2 \rho_{2,j} \frac{\overline{k}_{3,j}}{F_3}$$
(3.8c)

where

$$\overline{k}_{i,j} = \exp\left[\frac{-E_i}{1 + G\sum_{n=1}^{N+2} A_{(n)} x_j^{(n-1)}}\right]$$

In summary, there are (4N + 2) first-order differential equations of the form

$$\frac{df_{j}}{dt} = F_{j}(f_{1}, f_{2}, \dots, f_{Ne}, t)$$
 Ne = 4N + 2

resulting from (N + 2) equations for the temperature coefficients and N equations for each of the three solid species. The system of equations was then solved by a general integration subroutine which

employed an Adams-Moulton technique (10).

Initial Conditions

In order to start the solution for the (4N + 2) unknowns, their initial values must be given. The temperature approximation (3.5) is used in the boundary conditions (2.11) to obtain equations for the initial values of the coefficients.³

The initial temperature coefficients are

$$A_{(1)}(0) = 0 \tag{3.9a}$$

$$A_{(2)}(0) = 0$$
 (3.9b)

$$\sum_{n=3}^{N+2} A_{(n)}(0) x_{j}^{(n-1)} = 0 \qquad j=2,3,\ldots,N \qquad (3.9c)$$

$$\sum_{n=3}^{N+2} (n-1) A_{(n)}(0) = 1$$
(3.9d)

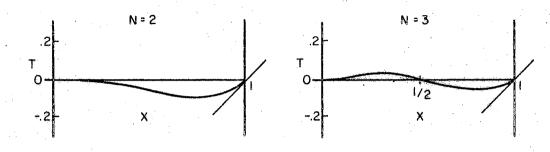
Using the values of x_{j} at each of the interpolation points in Equation (3.9c) produces (N - 1) equations which can be solved simultaneously with Equation (3.9d) for the N coefficients. The result, shown in Figure 6, is an (N + 1)-order polynomial for the initial temperature T = 0 that passes through zero at N points and has a slope of 1.0 at x = 1 and a slope of 0.0 at x = 0.

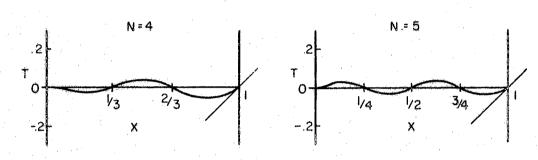
The initial species values are obtained directly by applying the boundary conditions (2.11b, c) at each x_i :

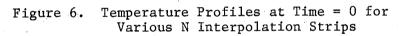
$$\rho_1(\mathbf{x}_1, 0) = 1$$
 (3.10a)

$$\rho_2(\mathbf{x}_j, 0) = \rho_6(\mathbf{x}_j, 0) = 0$$
 (j=1,2,...,N) (3.10b)

³See Appendix C.







Equations (3.5) through (3.10) constitute the complete problem to be solved on a digital computer to obtain temperature and species histories within the slab.

Summary of the Integral Technique

The purpose of the integral technique is to reduce the secondorder partial differential energy equation to a system of first-order ordinary differential equations that are then solved by an Adams-Moulton technique. The preceding discussion has described in detail the steps necessary to accomplish this and is now summarized.

First, divide the slab by N equally spaced planes parallel to the boundaries. Then, multiply the energy equation by a system of N linearly independent weighting functions that "weight" the heated boundary of the slab. Integrate the weighted equations over the slab to obtain N independent integral relations. Approximate the functions remaining under the integrals with polynomials in x having timedependent coefficients. Complete the integration to obtain a system of (N + 2) first-order ordinary differential equations for the timedependent temperature coefficients. Apply the solid species equations at the N strips to obtain 3N first-order ordinary differential equations. Use the boundary conditions to evaluate the initial values of the temperature coefficients and local solid species. Solve the system of (4N + 2) differential equations by an Adams-Moulton technique for the temperature coefficients (and thus the slab temperature distribution) and N local densities of each solid species at a given time.

Ã.

Computer Programming

The computer programs were developed to solve Equations (3.5) through (3.10) for the temperature profiles and local solid species concentration. From the calculated local solid species concentrations and their time derivatives, additional variables were calculated; the local amounts of each gas species produced from the N strips, the total amount of each gas species produced from the slab, the local rate of mass loss, the mass flux from the slab, and the average solid density for the slab,

The total amount of each gas species that was produced from each of the N strips was found from the solid species densities by^4

Gas₃ =
$$(1 - c_1) \left(\frac{\rho_6}{c_1 c_2} + \frac{\rho_6}{c_1} \right)$$
 (3.11a)

$$Gas_4 = 1 - \rho_1 - \frac{\rho_6}{c_1 c_2} - \frac{\rho_2}{c_1}$$
(3.11b)

$$Gas_5 = (1 - c_2) \frac{\rho_6}{c_2}$$
 (3.11c)

An Nth order polynomial in x having (N + 1) time-dependent coefficients was then used to approximate the average slab production of each gas species. The coefficients were evaluated by simultaneously solving the (N + 1) equations that are formed from applying Equation (3.11) at the N strips and the zero x-derivative of the solid species at x = 0. Integrating the approximations over the slab gives the total amount of each gas species produced from the slab. Likewise, the local values

⁴See Appendix E.

of the total solid species were used to obtain the average solid for the slab (called the Total Weight in the computer output).

The local gas generation rate was calculated by taking the negative of the sum of the local solid generation rates

 $-(\frac{d\rho_1}{dt}+\frac{d\rho_2}{dt}+\frac{d\rho_6}{dt}) \ .$

In the program, this value is called the Mass Loss Rate and is proportional to the reaction rate.

Finally, to find the mass flux out of the slab, a polynomial of (N + 1) terms was formed based on the local gas generation rates and the zero x-derivative of the generation rate at x = 0. Integrating the polynomial over the slab then gives the mass flux at the surface.

The programs that were written for the Integral technique solution are the MAIN program and subroutine DERFUN. A variable listing is given in Appendix G. Programs already in existence were used for the simultaneous solutions of the algebraic equations (subroutine SIMQ (16)) and the solution of the ordinary differential equations (subroutines START AND KAMSUB (10)). A description of the use of the latter three subroutines is given in Appendix H as well as a complete program listing of all routines in Appendix I. The basic steps followed in the computer programs are given in Table III with an expanded description given in Appendix J.

TABLE III

BASIC COMPUTING STEPS

- 1. Read the number of approximation strips, N.
- 2. Read the parameter values and the program control constants.
- 3. Calculate the initial values of the unknowns (Equations 3.9 and 3.10).
- 4. Calculate the 3N time derivatives of the local solid species (Equation 3.8).
- 5. Calculate the coefficients for the interpolation formulas (Equation 3.5).
- 6. Calculate the (N + 2) time derivatives of the temperature coefficients (Equations 3.6 and 3.7).
- 7. Integrate the (4N + 2) time derivatives to obtain values of the unknowns at the new time step.
- 8. Calculate the local values for the gas species, gas generation rate, temperature, and total solid.
- 9. Calculate the average slab values for the gas species, mass flux, and total solid.

CHAPTER IV

COMPUTER RESULTS

Introduction

Most of the computer work involved the development of both the integral technique and the computer technique as well as checking the accuracy of the results. However, this section will consider only some of the final results of the computer calculations.

Since a large number of parameters and possible parameter values were involved in addition to the various boundary conditions and reactions, a study was first made of the influence of individual parameters and events and then of the effects of combinations of events. The investigation of the influence of individual events was also necessary to the development of the computer technique since the selection of the program constants was highly dependent on the values of the problem parameters.

These results are not able to cover in depth all possible effects and combinations of the parameters but will show some of the major effects and trends for all the boundary condition cases and reaction combinations.

Parameter Values

Since the reaction scheme used in the program is based on organic solids such as cellulose and wood, the parameter values were chosen

from published data on these substances. As discussed earlier, Weatherford used the same physical and kinetic constants as Bamford, which represents values for one overall reaction in wood; their calculations were based on an activation energy of 33 kcal per mole and a frequency factor of 5 x 10^8 per second. Tang gives typical activation energies and frequency factors for cellulose in two temperature ranges; 23 and 54 kcal per mole for the activation energy and 0.3×10^6 and 0.6×10^7 per second for the frequency factor for temperatures from 280°C to 350°C. Murty and Blackshear also present kinetic constants which varied throughout cellulose from 13.6 to 22.6 kcal per mole for the activation energy and from 2 x 10^4 to 4×10^6 per second for the frequency factor. Similar values also apply to polymer foams as Tilley, et al, mention an activation energy of 50 kcal per mole and a frequency factor of 10^{13} per second for one of the major reactions in urethane foams. For three reactions, basic activation energies and frequency factors were chosen which were in the range of the reported values and which gave maximum reaction rates in the temperature ranges reported by Broido.

The overall heats of reaction for wood and cellulose are of the same magnitude with wood being exothermic and cellulose, endothermic. Weatherford used -86 cal per gm for wood while Tang reported $\frac{+88}{48}$ cal per gm for cellulose. Broido describes the three major reactions of cellulose as slightly endothermic, highly endothermic and highly exothermic. For the three reaction cases, basic heats of reaction were chosen which demonstrated Broido's findings and had an overall heat of reaction similar to the published values.

A basic value for the radiant heat flux was chosen as the maximum

initial heat flux in Weatherford's experiments. While experimental radiant heat fluxes are normally much higher than Weatherford's initial convective value, the slab thicknesses used are much smaller. Therefore, the numerator, qL, of the heat flux parameter, G, would be of the same magnitude in either case.

The dimensional values used in computing the basic parameters are given in Table IV, with the resulting non-dimensional parameters given in Table V. The particular parameter values used in each computer run given in the results are presented in Table VI.

The Inert Slab

Inert slab calculations were made and compared with known solutions. Carslaw and Jaeger (6) give the exact solution for an inert slab heated by a constant heat flux. As shown in Figure 7, increasing the number of approximating strips gives a more accurate answer, and, a good approximation for the inert case was obtained with only three approximating strips. Increasing the number of approximating strips results in curves which lie midway between the exact solution and the previous approximation.

For the convective boundary condition, Weatherford (23) graphs the solution for the surface temperature of an inert slab. Figure 8 shows that three approximating points again give the exact solution except for the early times. Higher approximations are practically indistinguishable from the exact solution for all times and for the chosen parameters.

Thus, the exact solutions show that the integral technique can accurately represent the linear portion of the problem. However,

TABLE IV

BASIC DIMENSIONAL CONSTANTS

a1	$8.2 \times 10^4 \text{ sec}^{-1}$
a2	8.2 x 10^6 sec ⁻¹
a3•••••	$8.2 \times 10^9 \text{ sec}^{-1}$
c_1 (one reaction)	0.5 gm/gm
c_1 (three reactions)	0.9 gm/gm
c ₂	0.833 gm/gm
C	0.55 cal/gm-°K
E ₁ • • • • • • • • • • • • • • • • • • •	23,844 cal/mole
E ₂ • • • • • • • • • • • • • • • • • • •	31,000 cal/mole
E ₃	35,748 cal/mole
h • • • • • • • • • • • • • • • • • • •	4.4 x 10 ⁻⁴ cal/cm ² -sec-°K
K ₀ • • • • • • • • • • • • • • • • • • •	2.7 x 10^{-14} cal-cm/cm ² -sec-°K
L	1.0 cm
Q_1	+16.5 cal/gm
Q ₂	+165 cal/gm
Q3 • • • • • • • • • • • • • • • • • • •	-94.2 cal/gm
q ₀	0.081 cal/cm ² -sec
R°	1.986 cal/mole-°K
T ₀	300°K
ρ ₀	0.6 gm/cm^3

TABLE V

BASIC NON-DIMENSIONAL PARAMETERS

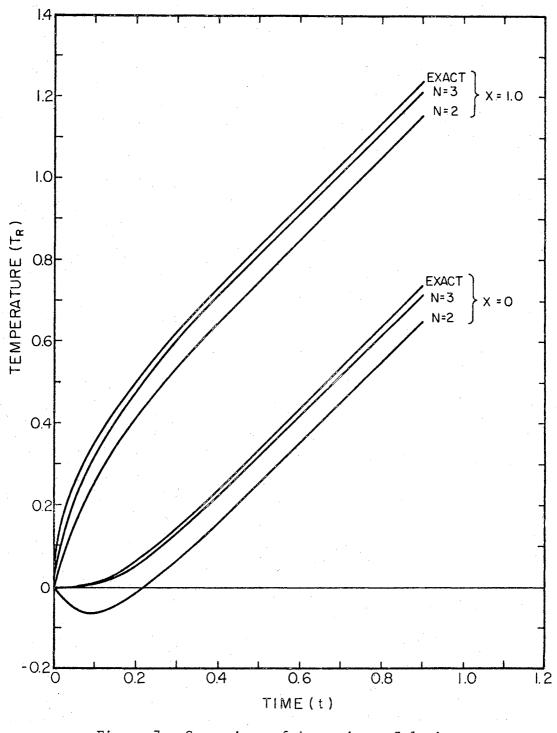
Nu			• • • • • • • • •	2.0
E ₁	F • • • • •	• • • • • • •		40.0
E ₂		• • • • • • •	••••••••	52.0
E ₃				60.0
F ₁				0.1×10^{-7}
F ₂		• • • • • • •	• • • • • • • • •	0.1×10^{-9}
F ₃	• • • • •		• • • • • • • • •	0.1×10^{-12}
G (Type I)	• • • • • • •		•••••	1.0
G (Type II)	• • • • • •	• • • • • • •	• • • • • • • • •	1.535
R_1		· · · · · · ·		+0.1
R ₂	• • • • • •		••••••	+1.0
R ₃				-0.571

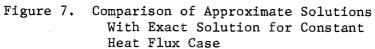
TABLE VI

PARAMETER VALUES USED IN COMPUTED RESULTS

						,										
Run <u>Number</u>	<u>B.C.</u>	Case	<u>E1</u>	E ₂	<u>E3</u>	F ₁	F ₂	F3	R ₁	<u>R2</u>	R ₃	<u>c</u> 1	c 2	G	Nu	
210	0	1	40			10-8			0			• 5	¹	1.0		
211	0	1	40	<u> </u>		10 ⁻⁸			0			.8		1.0		
213	1	2	40			10-8			0			.5		1.535	2.0	
220	· 1	1	40			10~8			0			.5		1.535	2.0	
222	0	1	40			10-8			+.3			.5		1.0		
223	0	0	40			10-8			*. 3			.5	·	1.0		
226	0	0	40			10-8			3			.5		1.0		
228	0	0	40			10-8		 '	0			.5		1.0		
229	0	0	40			10-8			0			.5		2.0		
243	0	0	40			10-8			+.3			.5	· '	1.0		
261	0	1	40	52	60	10-9	10-11	10-13	+.1	+1.0	571	.9	.833	1.0		
262	1	0	40	52	60	10-8	10-10	10-12	+.1	+1.0	571	.9	.833	1.5	2.0	
263	1	0	40	52	60	10-8	10-10	10-12	÷.1	+1.0	571	.9	.833	1.75	2.0	
264	1	0	40	52	60	10-8	10-10	10-12	+.1	+1.0	571	.9	.833	1.45	2.0	
265	1	2	40	52	60	10-8	10-10	10-12	+.1	+1.0	571	.9	.833	1.75	2.0	
080	1	0	50	55.66	60	10-10	10-11	10-12	0	0	0	.9	.833	1.535	2.0	
081	1	0	50	55.66	60	10-10	10-11	10-12	0	0	0	.9	.833	2.0	2.0	
082	1	0	48	55.66	60	10-10	10-11	10-12	0	0	0	.9	.833	1.535	2.0	
083	1	0	50	55.66	60	5x10-10	10-11	10-12	0	0	0	.9	.833	1.535	2.0	
125	0	0	50	55.66		10-10	10-11		0	0	0	.5		1.0		
131	0	0	50	55.66		10-10	10-11		0	0	0	.5		1.5		
143	0	2	55.66			10-10			0	0	0	. 5		1.0		

Note: For the constant heat flux boundary condition, B.C. = 0. For the convective heated boundary condition, B.C. = 1. Exothermic reactions denoted by (-) and endothermic reactions by (+).





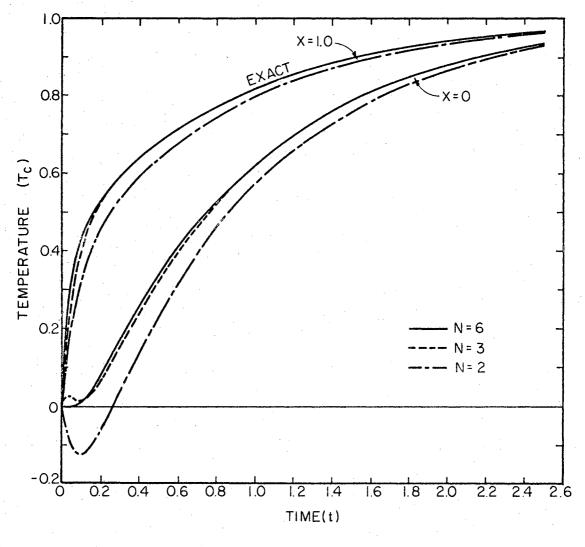


Figure 8. Comparison of Approximate and Exact Solutions for Convective Heating, Nu = 2

there are no exact solutions for checking the validity of the technique when including the non-linear terms for the generated heat and the change in effective thermal conductivity. For this reason, the method of computation was made to allow any order of approximation to be used, enabling the results to be generally checked by comparing with a higher approximation.

One Reaction--Case 0

In order to interpret the results with multiple reactions, the single reaction cases were studied first. Initially, a constant effective thermal conductivity was assumed.

In Figure 9 the temperature history of the front and back surfaces are shown for a slab heated by a constant heat flux and an exothermic reaction. The front surface temperature follows the inert value until the reaction begins to generate heat, causing the temperature to increase at a higher rate. As the reaction proceeds into the slab, the amount of generated heat increases, thereby increasing the amount of conducted heat. This has two major effects: temperatures within the slab begin to deviate from the "inert" value even before a reaction starts at that particular location, and the temperature rise becomes increasingly steeper at greater depths.

As the reaction ends at the heated surface, the temperature increases at a constant rate that is higher than the inert value, showing that the rate of heat conduction into the slab is less due to the increased internal temperatures. A result of the accumulation of generated and conducted heat is that the temperature at the back surface (adiabatic surface) becomes higher than the slab just ahead of it for

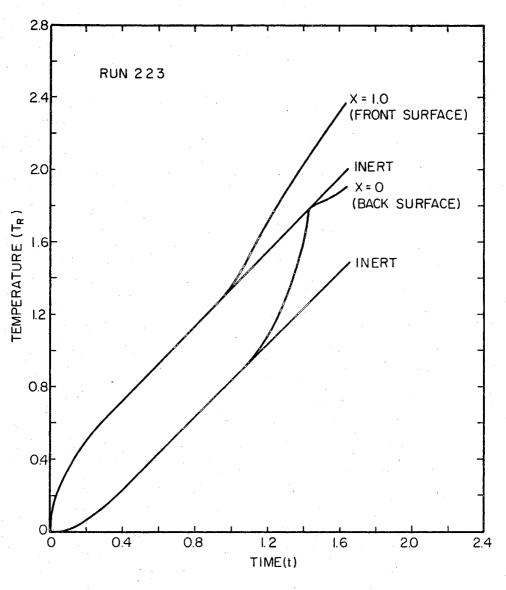


Figure 9. Effect of Heat of Reaction: Temperature History of Front and Back Surfaces--Constant Heat Flux, One Exothermic Reaction, Case O

a short time, up until the reaction reaches its maximum. An abrupt drop in the rate of temperature rise then follows as the reaction in the slab ends. When the reaction ceases, the rate of temperature rise, compared to the inert solution, ranges from higher at the front surface to lower at the back surface. Of course, after this transient period, the temperatures throughout the slab would become parallel to the inert solution with an offset from the inert values.

Results for an endothermic reaction are shown in Figure 10. Effects are similar to the exothermic reaction except with an opposite sign. Note that the reaction events are spread over a longer time period, and the temperature rise becomes parallel to the inert solution very quickly after the slab is through reacting.

Another basic difference in the character of endothermic and exothermic reactions can be seen in Figures 11 and 12 where temperature profiles are given as the reaction proceeds through the slab. This time the endothermic reaction is discussed first. The profiles all have a slope of 1.0 at the front surface and zero at the back surface. The earliest profile shows the temperature distribution when the front surface reaction starts. The next profile (t = 1.33) was taken when the surface reaction rate is at its maximum. The effect of the energy absorbed by the reaction is to depress the temperature rise, i.e., the front temperature rise is less than the back surface. Likewise, as the reaction proceeds through the slab, the profiles show the same effect with the last profile showing that the reaction is at the back surface.

For the exothermic reaction, however, the profiles are warped because of the reaction, as seen in Figure 12. The increasing times

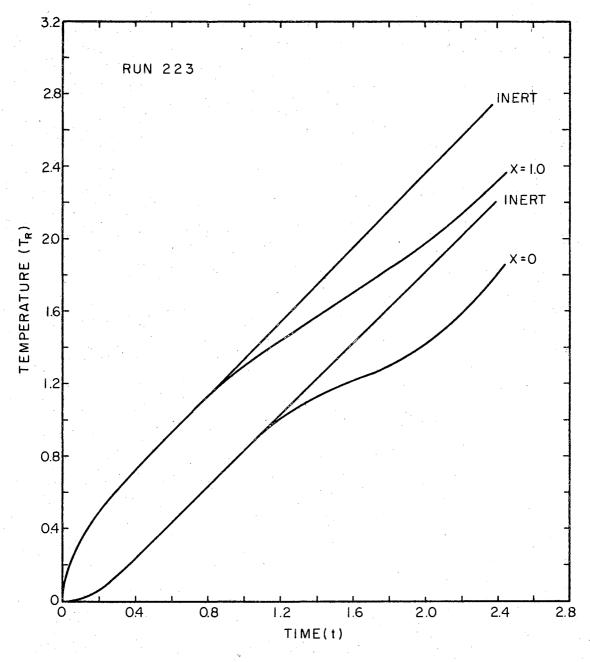


Figure 10. Effect of Heat of Reaction: Temperature History of Front and Back Surfaces--One Endothermic Reaction, Constant Heat Flux, Case 0

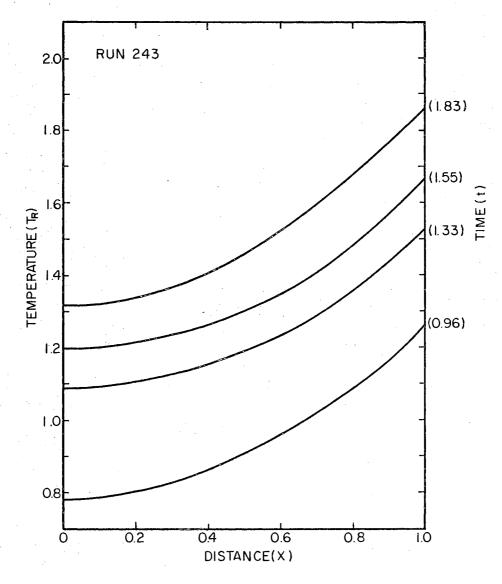
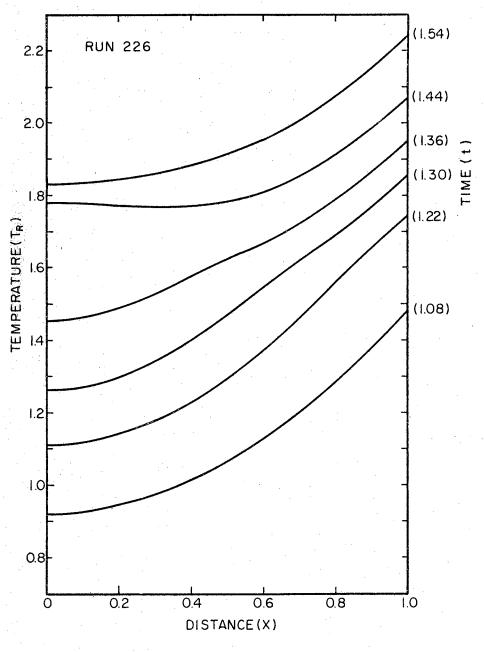
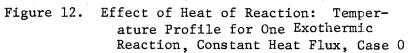


Figure 11. Effect of Heat of Reaction: Temperature Profiles for One Endothermic Reaction--Constant Heat Flux, Case O





trace the reaction through the slab with the back part briefly achieving a higher temperature than the middle as described earlier. Because of the profiles warping, a larger number of approximating strips is needed for exothermic reactions than for an equivalent endothermic reaction.

The reaction rate shows another striking difference between endothermic and exothermic processes. Figure 13 demonstrates that the maximum reaction rate increases for exothermic reactions at distances farther into the slab. The curves show the progression of the reaction through the slab and show the reaction rate distribution at six times. These are the times when the reaction rate reached a maximum at each of the six interpolation strips. The maximum reaction rate at the back surface is more than two and one-half times the rate at the front sur-This is a result of the higher temperature rise from the inface. creasing amount of heat conducted ahead of the reaction. Since the rate of reaction is dependent on the product of the amount of reactant and the temperature-dependent rate constant, the additional conducted heat causes the temperature to be slightly higher for a given amount of reactant, thus increasing the reaction rate. This is shown in Figure 14 where the back surface always has a higher temperature than the front surface for a given amount of reactant.

Of course, the increase in the reaction rate through the slab also means that the heat generation rate increases. For reactions having higher heats of reaction, more heat would be generated causing the difference in the reaction rates to be greater. Finally, by comparing the times at which the maximum rate occurred at each x-position, it is seen that the velocity of the "reaction zone" increased as it

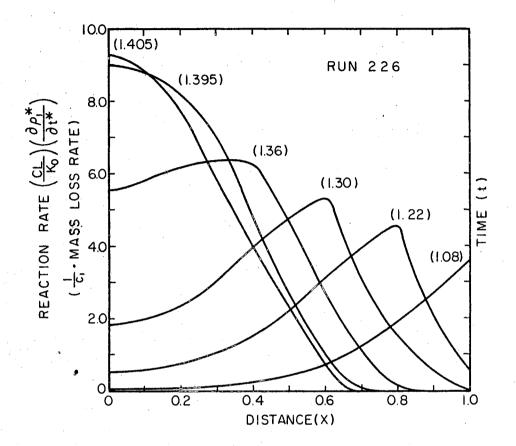
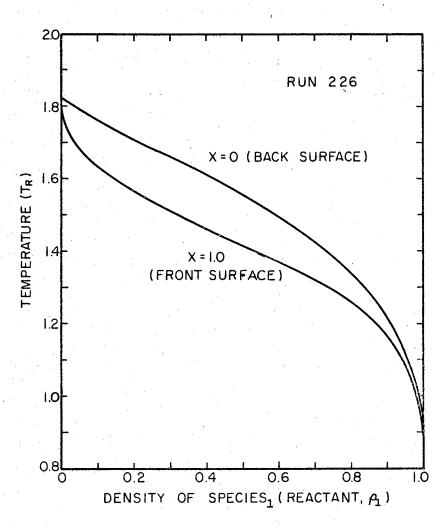
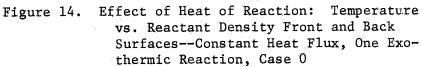


Figure 13. Effect of Heat of Reaction: Reaction Rate in Slab for One Exothermic Reaction--Constant Heat Flux, Case O





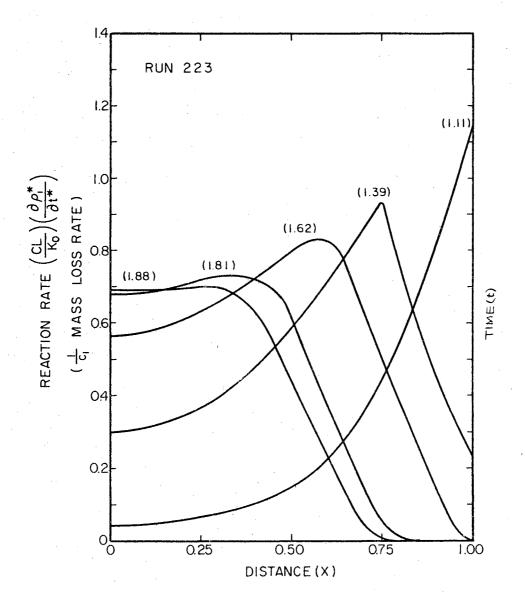
proceeded into the slab. Two factors contribute to this: the increasing rate of reaction and the longer residence times at greater depths resulting in less solid remaining for the reaction.

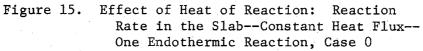
For the endothermic reaction, Figure 15 shows that the maximum reaction rate decreases from the front to the back surface. This indicates that a result of the energy absorption is a lower temperature for a given reactant density. The difference in times at which the maximum reaction rate occurred again shows that the "reaction wave" accelerated as it traveled farther into the slab. Thus, the longer reacting times for the interior solid contributed more to increasing the velocity of the "reaction wave" than the decreasing reaction rate did to retard it.

25

A difference of major importance in the effects of the endothermic and exothermic reactions is shown by the mass flux curves in Figure 16. The area under the mass flux curves equals the total mass of solid lost from the slab and is the same in both cases. The exothermic reaction was completed in half the time of the endothermic reaction, and the maximum mass flux was increased by three times. The "lump" on the side of the endothermic curve near the end of the reaction is apparently due to the increasing reaction rate near the insulated surface.

Figure 17 gives the solid distribution for both reaction cases as points in the slab reached a given density. The solid distribution profiles were steeper for the exothermic reaction than for the endothermic reaction up until the reaction was nearly completed. This is a result of the longer residence times of the solid interior for the endothermic reaction.





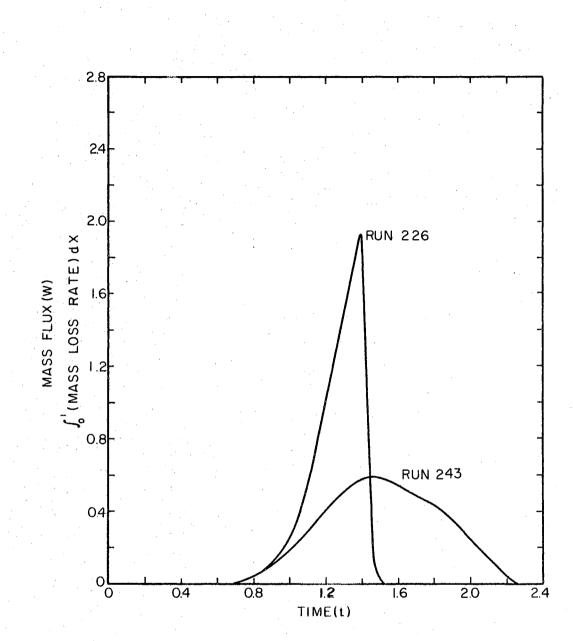
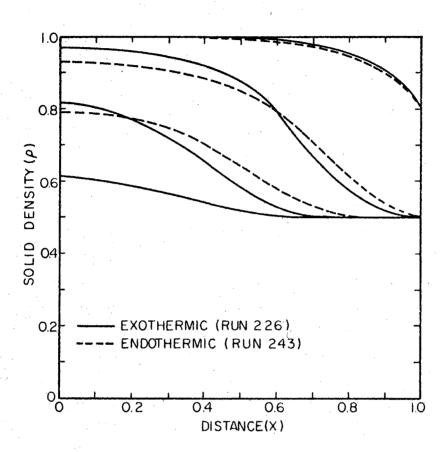
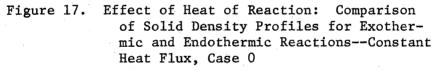


Figure 16. Effect of Heat of Reaction: Mass Flux From Slab for Exothermic and Endothermic Reactions--Constant Heat Flux, Case 0





The previous discussion has been concerned with the differences in endothermic and exothermic reactions under the same heating conditions. The effect of a change in the heat flux parameter alone can be seen by making the heat of reaction parameter zero. Figure 18 gives the reaction rate vs. surface temperature for two difference values of G. Not only does increasing the heat flux cause an increase in the reaction rate but it also causes the maximum reaction rate to occur at a higher temperature. Since a given temperature fixes the reaction rate constant, the increase comes from a larger value for the density of reactant (see Equation 2.10a). This is verified by Figure 19 which shows the increase in the reactant present at a given temperature. This effect is a result of a higher rate of increase in the reaction temperature with increasing G, giving the solid less time to react, thus making more solid available at a given temperature.

In summary, the computations have given temperature profiles and histories for exothermic and endothermic reactions with constant parameters. The heat generated or absorbed from the reactions have been shown to have a great effect on the solid ahead of and behind the reactions, not only with respect to temperatures, but also reaction rates and mass flux. Increasing the heat flux parameter was shown to increase the maximum reaction rate and also the temperature at which it occurs.

One Reaction--Cases 1 and 2

Temperature histores were calculated to show the effect of a variable effective thermal conductivity. A density-dependent, effective thermal conductivity affects both the front surface boundary condition

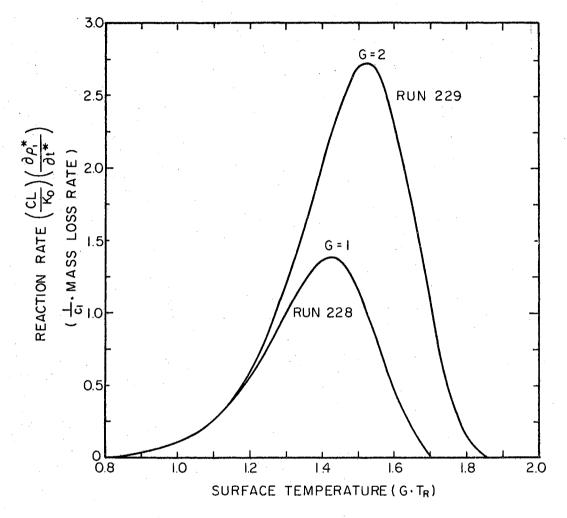
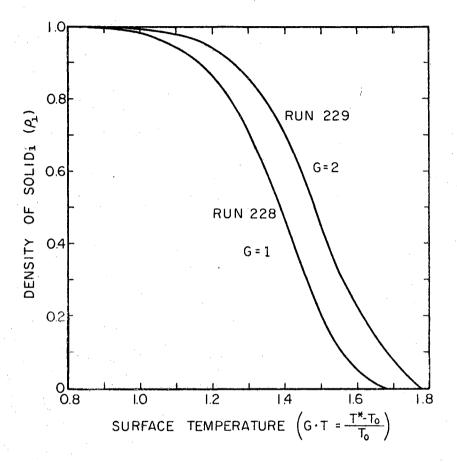
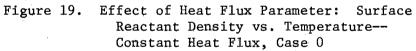


Figure 18. Effect of Heat Flux Parameter: Surface Reaction Rate vs. Temperature--Constant Heat Flux, Case 0





and the energy equation for Case 1 or just the energy equation for Case 2.

Figure 20 shows, for the convective heating case, the surface temperature history for Cases 1 and 2 and also the constant effective thermal conductivity Case 0. The heat of reaction parameter is zero so the temperature deviations from the constant conductivity case occur only from the changes in the effective thermal conductivity and the heat transfer coefficient. For Case 1 the heat transfer coefficient is constant while the effective thermal conductivity decreases as solid is lost. The result is that the rate of heat conducted from the surface is less, causing the temperature to increase.

For Case 2 the heat transfer coefficient and the effective thermal conductivity both decrease as solid is lost, with the result that the surface boundary condition is the same as the constant effective conductivity case while the decreasing effective conductivity appears as an equivalent endothermic contribution in the energy equation. Neither case causes the temperature to deviate much from the constant effective conductivity solution for the parameters that were used. The primary reason is that the temperature dependency in the surface boundary condition causes a compensating effect in both cases. For Case 1, as the surface temperature tends to rise, the difference in the source and surface temperatures becomes less, decreasing the heat flux to the surface and partially decreasing the rate of increase of the surface temperature. Likewise, for Case 2, as the surface temperature tries to rise at a slower rate, the temperature difference is greater causing a higher heat flux. As higher source temperatures are used, the trend would be for a greater effect of the changing effective thermal

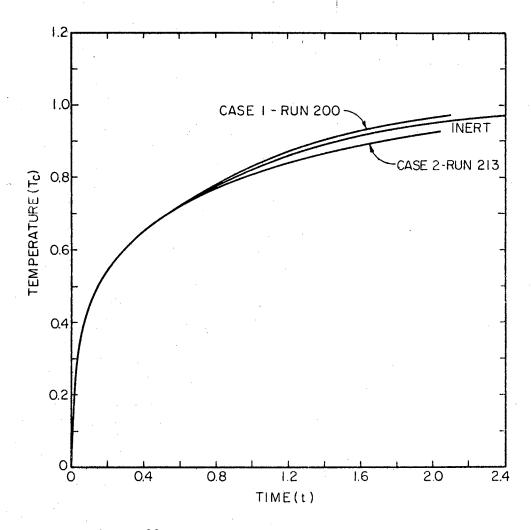


Figure 20. Effect of Variable Effective Thermal Conductivity and Heat Transfer Coefficient: Temperature History of Front Surface, Convective Boundary Condition, Cases 1 and 2

conductivity on the temperature since the surface and source temperature difference would increase, thus decreasing the influence of a changing surface temperature.

When the radiation boundary condition is used, the effects are more pronounced. For no heat of reaction and a constant heat flux input, Figure 21 shows how the decreasing effective thermal conductivity affects the surface temperature history (Case 1). Since the effective conductivity depends on the amount of solid present, calculations were made for two final solid densities, 50% and 80% of the original density. The temperature increases faster as the amount of final solid decreases. The slope of the temperature rise after the reaction has been completed is the inverse of the final density. As the solid drops from its original density to 50%, the surface boundary condition shows that the slope of the temperature profile increases from 1 to 2. For 80% of the original solid density, the slope increases from 1 to 1.25. Thus, for Case 1, low final solid densities will greatly affect the slab temperatures.

Case 2 for the constant heat flux condition has essentially the same boundary conditions as the constant effective thermal conductivity case but includes the effect of the change in the effective conductivity in the energy equation. Figure 22 gives the temperature history for the front and back surfaces with no heat of reaction. The result shows that the changing effective thermal conductivity appears in the energy equation as an endothermic reaction if the heat input decreases so as to hold constant the slope of the surface temperature profile. This contribution does not occur at exactly the same time as a true endothermic reaction because the heat generated from a reaction depends

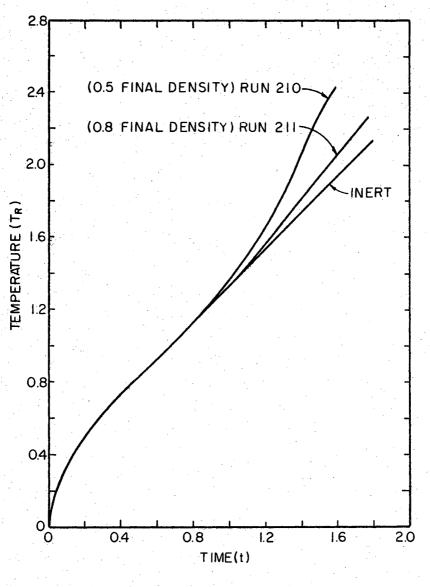


Figure 21. Effect of Solid Density on Effective Thermal Conductivity: Temperature History of Front Surface for Final Solid Densities of 0.5 and 0.8--Constant Heat Flux, Case 1

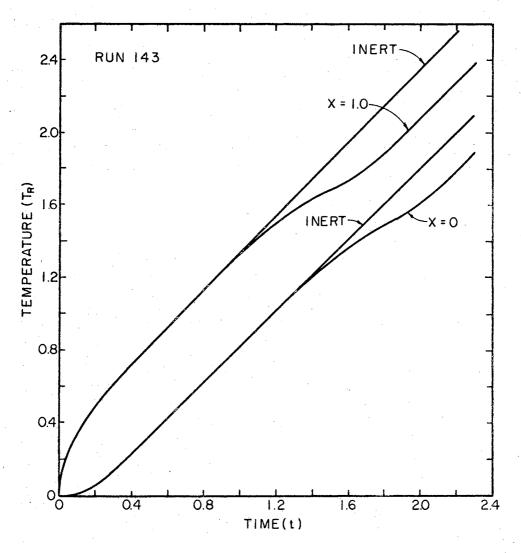


Figure 22. Effect of Conductivity Change on Energy Equation: Temperature History of Front and Back Surfaces for Constant Heat Flux--Case 2

on the time rate-of-change of the solid, while the contribution from the variable conductivity depends on the product of the rates of change of density and temperature with respect to x (see Equations 2.3, 2.7, and 2.8).

The combination of an endothermic reaction and a changing effective thermal conductivity is given in Figure 23. This figure shows the surface temperature history for Cases 1 and 0 together with Case 1 having an endothermic reaction. For Case 1 with the reaction, the rate of temperature rise first begins to decrease from the constant effective conductivity case, indicating that the endothermic reaction is affecting the temperature more than the changing effective conductivity. As the reaction proceeds, the high rate of increase of the temperature from the changing conductivity eventually causes the temperature to increase at a greater rate until the surface reaction ends and the rate of temperature rise is parallel to the Case 1 value. The overall effect of the changing effective conductivity cannot be represented by a single equivalent reaction.

The effect of a density-dependent, effective thermal conductivity and heat transfer coefficient is negligible for convective heating when source temperatures are low. For radiant heating and high source temperature convective heating, the change in effective thermal conductivity can greatly affect the temperature history of the slab. This is especially true as the final densities become smaller. These effects must be carefully considered when interpreting temperature histories.

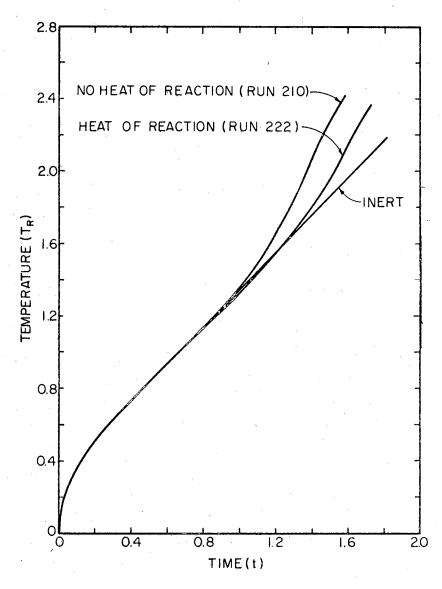


Figure 23. Effect of Heat of Reaction: Temperature History of Front Surface for Constant Heat Flux--Case 1, With and Without Heat of Reaction

Three Reactions--Case 0

The effects of parameter changes on the competing reactions were studied with zero heats of reaction and a constant effective thermal conductivity. Figure 24 shows typical density curves for the three solid species and Gas_4 . Gas_3 and Gas_5 are not shown since they are similar to Gas_4 . As Solid₁ reacts, it gives off Gas_3 and forms Solid₂. At higher temperatures the competing reaction tends to form more Gas_4 from Solid₁. As the temperature rises, Solid₂ begins to give off Gas_5 and form Solid₆. When the rate of reaction of Solid₂ equals its rate of formation, it species curve reaches a maximum. The final species present is Solid₆ with Gas_4 indicating the total amount produced from the slab.

For cellulose, Gas_4 is the flammable species that competes for Solid₁. Its importance in ignition and combustion occurs in several ways; ignition may occur if the surface temperature and the quantity of Gas_4 evolving is high, while sustained combustion depends on the rate as well as the quantity of Gas_4 evolving. Figure 25 indicates that a change in the heat flux parameter does not have as much effect on the total amount of Gas_4 produced as it does on the rate of evolution and the time when it is evolved. The maximum rate of production of Gas_4 (indicated by the slope of the curves) is not much greater for the higher heat flux parameter case but the maximum is attained quicker and held throughout most of the reaction.

The amount of Solid₁ that does not form Gas_4 has gone to produce the other species. Since a higher heat flux produces more Gas_4 , less Solid₁ is available for the other species. Figure 26 shows that the amount of char formed is less for the higher heat flux but like Gas_4 ,

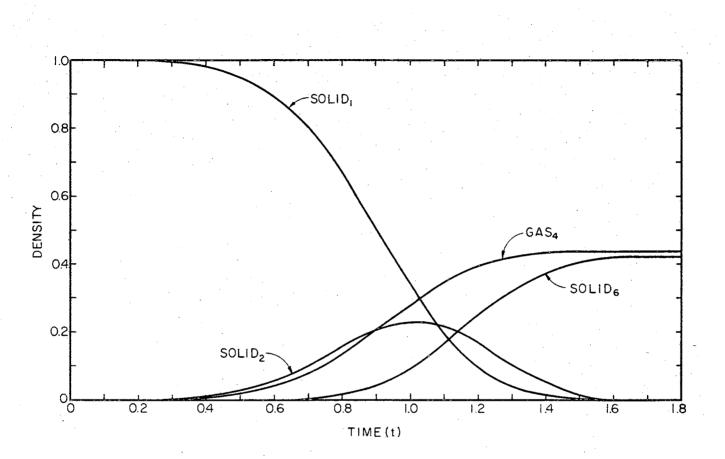


Figure 24. Selected Species Histories for Three Reactions--Convective Heating, Case O

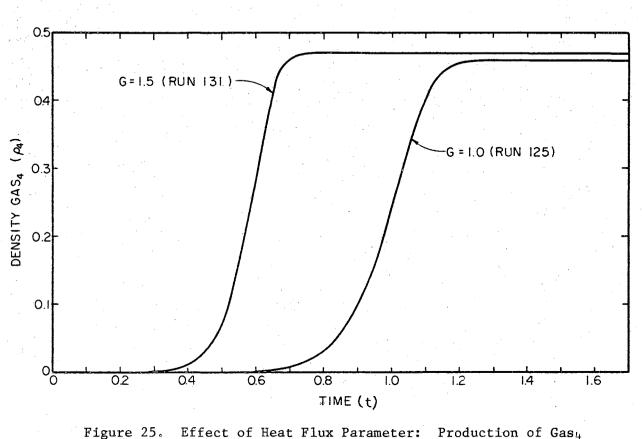


Figure 25. Effect of Heat Flux Parameter: Production of Gas₄ from x = 1.0--Constant Heat Flux, Three Reactions, Case 0

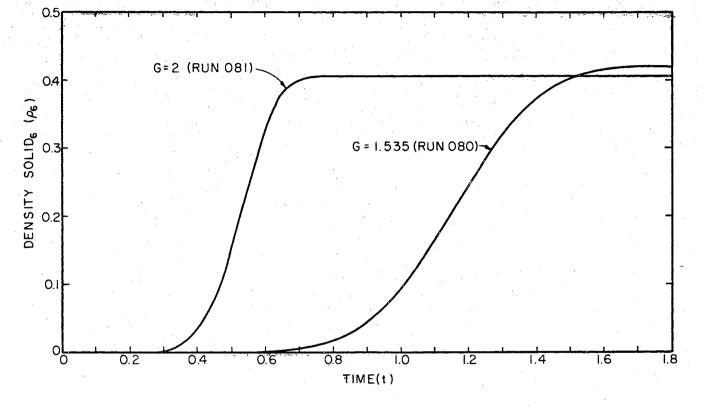


Figure 26. Effect of Heat Flux Parameter: Production of $Solid_6$ at x = 1.0--Convective Heating, Three Reactions, Case 0

its rate of formation is higher and time of formation is much sooner than for the lower heat flux case.

One theory for the addition of chemicals to the solid for fire retardant purposes is that it tends to lower the activation energy for the first reaction thus producing more char and less Gas₄. Figure 27 shows that lowering the activation energy for reaction 1 by 4% increased the amount of char by 30%, which also greatly reduced the amount of Gas₄ produced. The reaction times were about the same with the rate of production then depending on the amount of Solid₆ produced.

The largest experimental error in the kinetic constants occurs in the determination of the frequency factor since the actual mass of reactant is difficult to determine for multiple reactions. Figure 28 shows the effect of a change in the frequency factor for reaction 1 on the amount of Solid₆ produced. Increasing the frequency factor by a factor of 5 increased the amount of char by a factor of 3.

The amount of solid present may be important to slab strength considerations as well as for thermal conductivity effects. Figure 29 shows the total solid history for changes in the activation energy of reaction 1. The early times of the reaction show less solid present for the lower activation energy case and the latter times show more solid present. The higher activation energy case delays the reaction of Solid₁, causing more total solid to be available in the early times. In fact, combinations of the kinetic constants could produce a crossover point of equal amount of solid for high and low activation energies that was less than some value for strength requirements. This would mean that addition of fire retardant chemicals could produce a solid that burned slower but failed quicker than an untreated solid.

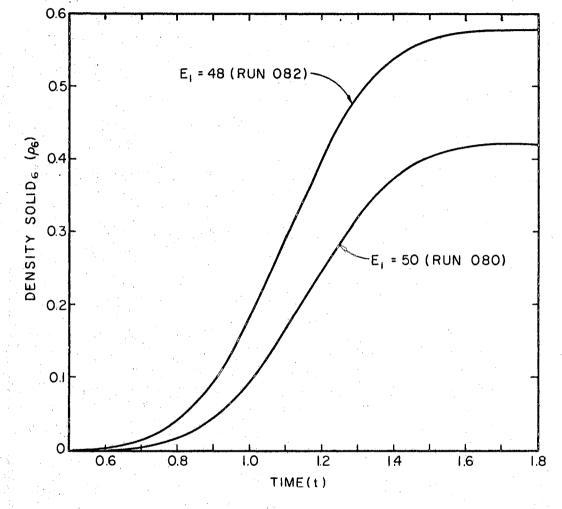
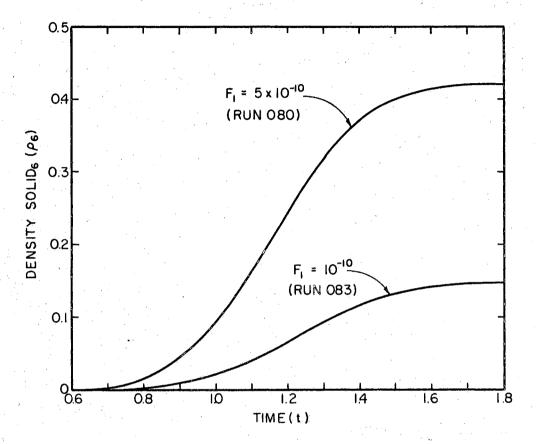
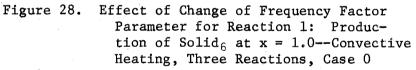


Figure 27. Effect of Change of Activation Energy Parameter for Reaction 1: Production of Solid₆ at x = 1.0--Convective Heating, Three Reactions, Case 0





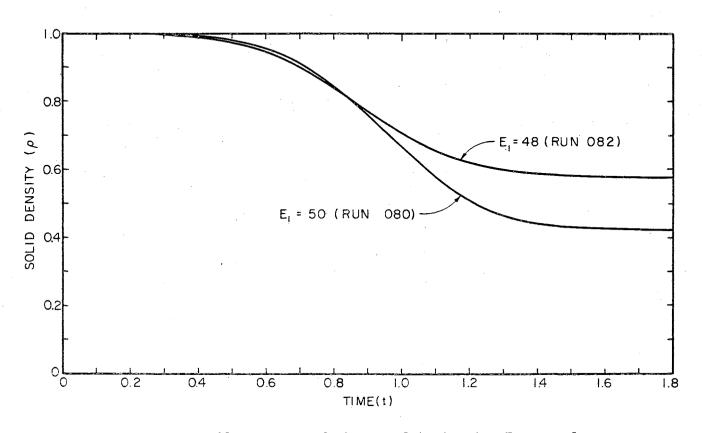


Figure 29. Effect of Change of Activation Energy of Reaction 1: Production of Total Solid at x = 1.0--Convective Heating, Three Reactions, Case 0

In conclusion, this section showed how the competing reactions affected the production of $Solid_6$ and Gas_4 when changes were made in the kinetic constants and heat flux parameter.

Three Reactions with Heats of Reaction

After investigating the influence of separate events, the problem was calculated with all the reactions and heats of reaction included. For the radiantly heated boundary condition, the density-dependent, effective thermal conductivity case was used with a constant heat flux at the surface and no heat losses (Case 1). Cases 0 and 2 are not presented since Case 1 is closer to the actual physical situation, and similar effects of the two cases are shown by the convective heated boundary conditions.

Figure 30 gives the temperature history for the radiantly heated slab with endothermic, competing reactions producing Gas_3 , $Solid_2$, and Gas_4 , and an exothermic reaction producing Gas_5 and $Solid_6$. At the surface, the endothermic reactions and the decreasing effective thermal conductivity tend to cancel the heat production effects while the endothermic reactions and the decreasing conducted heat effects the temperatures farther into the slab. Eventually, the exothermic reaction at the surface combines with the decreasing effective thermal conductivity to increase the temperature from the inert solution. An effect due to the approximation technique is seen next as the surface temperature tends to level off when the exothermic reaction ends and the interior endothermic reactions continue. When the exothermic reaction starts at the next approximation point (x = 0.8), the temperatures all increase at a steep rate as the combined effects of heat

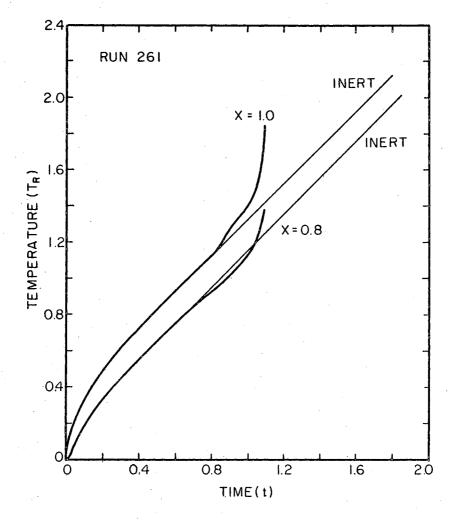
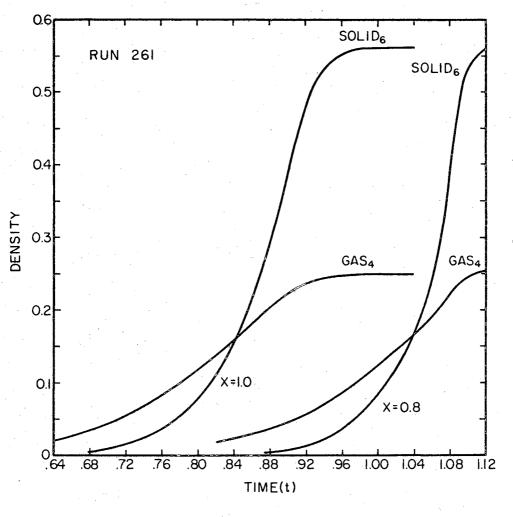
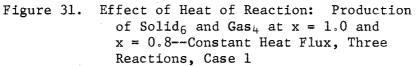


Figure 30. Effect of Heat of Reaction: Temperature History at x = 1.0 and x = 0.8 for Constant Heat Flux--Three Reations with Heats of Reactions, Case 1

generated and the decreasing effective thermal conductivity force the temperature up. In reality, the surface temperature would continue increasing when its exothermic reaction ended for the reaction would be continuing into the slab, decreasing the heat conducted from the surface. Figure 31 shows the production of Solid₆ and Gas₄ at x = 1.0and x = 0.8 (the next approximating strip). The slopes of the species curve is proportional to the rate of the reaction producing that species. The rate of reaction 2 (producing Gas4) begins increasing at x = 0.8 before it starts decreasing at x = 1.0. Thus, the finite distance between the approximating strips does not visibly affect the temperature history. The rate of reaction 3 (producing Solid₆) however, is almost zero at x = 1.0 before it becomes significant at x = 0.8. Therefore, the effect of the finite distance is as though the reaction was turned off and then on. This shows how the width of the reaction zone can affect any approximation technique that uses a finite number of points to represent the slab. Without changing the parameters or kinetic constants, this error can be reduced by increasing the number of approximating strips and thus decreasing the distance between the strips. This also illustrates the difficulty in calculating the temperatures for this type of boundary condition with exothermic reactions because, as the reaction proceeds into the solid, the reaction rate increases, the temperatures increase, and the higher temperature reactions (here, the exothermic reaction) dominate, causing the temperatures to increase more, increasing the mass loss and decreasing the effective thermal conductivity. This quickly becomes a runaway situation with the slab probably following different boundary conditions after a short period of time due to re-radiation or ignition. Of





course, selected values of the reaction constants are possible that would enable this case to be calculated accurately but would probably not be physically realistic.

For the convectively heated boundary condition, three reactions are used, first with a constant effective thermal conductivity and then with the Case 2 conditions. Figure 32 gives the temperature history of the front and back surfaces. The initial endothermic reactions cause the surface temperature to fall below the inert solution until the exothermic reaction increases the temperature above the inert value. At the back surface, the decreasing conducted heat and the endothermic reactions cause the rate of temperature rise to become less than the inert solution. However, as the exothermic reaction begins generating heat from the front surface, the heat begins to accumulate and increases the reaction rate within the solid and finally causes the back surface temperature to exceed the front surface temperature. The initial endothermic character of the reactions caused the competing reactions to produce more Solid₂, and thus more Solid₆, part of the way into the solid. As the exothermic reaction began to dominate, the competing reactions began producing more Gas_4 , and thus, less Solid₆. Table VII shows the final slab distribution of $Solid_6$.

If the heat flux parameter is decreased slightly from 1.5 to 1.45, then the first competing reaction will produce more Solid₂, and thus Solid₆, by being held longer at the lower temperatures. Figure 33 shows the difference in the temperature histories of the front and back surfaces for the two heat flux cases. The smaller heat flux delays the heating events and, by producing more Solid₆, causes the exothermic reaction to produce more heat and increase the slab temperatures

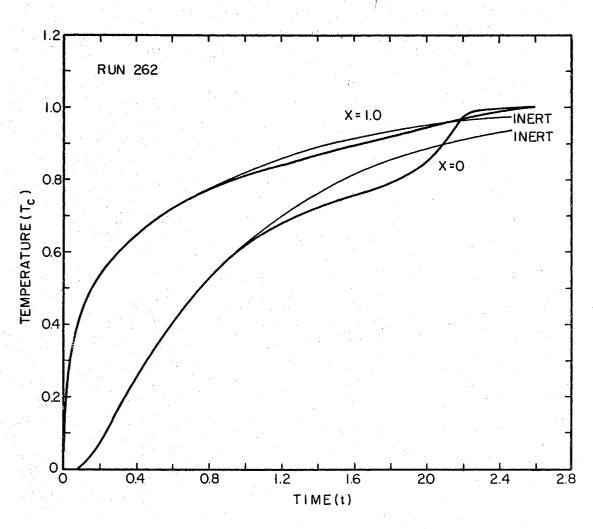


Figure 32.

Effect of Heat of Reaction: Temperature Histories of Front and Back Surfaces for Convective Heating, Three Reactions with Heats of Reaction, Case 0, G = 1.5

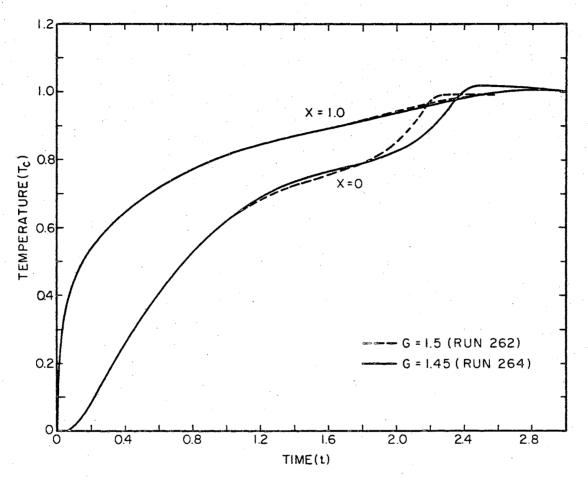


Figure 33. Effect of Heat Flux Parameter: Temperature Histories of Front and Back Surfaces--Convective Heating--Three Reactions with Heats of Reaction, Case O

TABLE VII

Density of Solid₆ <u>x</u> G = 1.45G = 1.5G = 1.751.00 .5248 .5195 .4975 0.75 .5260 .5209 .5005 0.50 .5241 .5193 .5008 0.25 .5209 .5161 .4984 0.00 .5192 .5143 .4965

SLAB DISTRIBUTION OF SOLID6

at the later times.

The mass flux also reflects the change in the heat flux parameter. Figure 34 shows that the higher heat flux not only causes the mass flux to increase and occur sooner, but also tends to separate the two main gas producing events which occur as a result of the overall reaction rate increasing in the slab after initially decreasing.

The higher heat flux was then used with the Case 2 boundary condition that considers a density-dependent, effective thermal conductivity and heat transfer coefficient. Figure 35 gives the temperature histories for the two surfaces compared with the Case 0 solution. The resulting effect is that of an endothermic reaction that occurs over the entire temperature range but occurs slightly "out-of-phase" with the other reactions (as described earlier). Also the exothermic reaction tends to be dampened since the trend to increase the change in demsity and temperature in the x-direction would give a higher endothermic

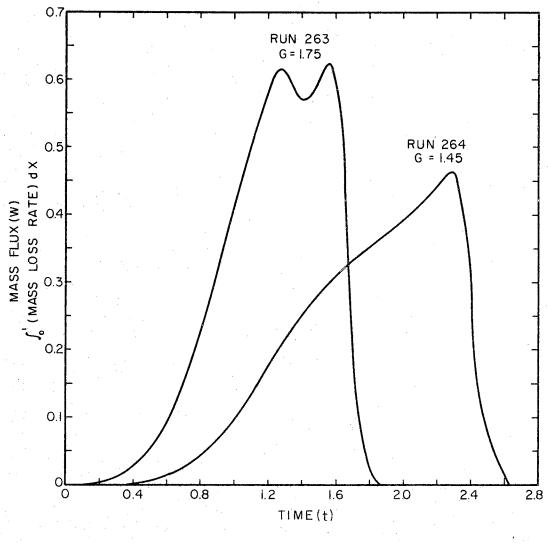


Figure 34. Effect of Heat Flux Parameter: Mass Flux . from Slab--Convective Heating, Three Reactions, Case 0

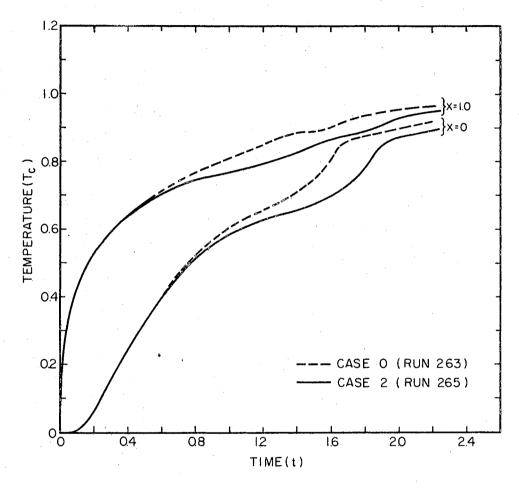
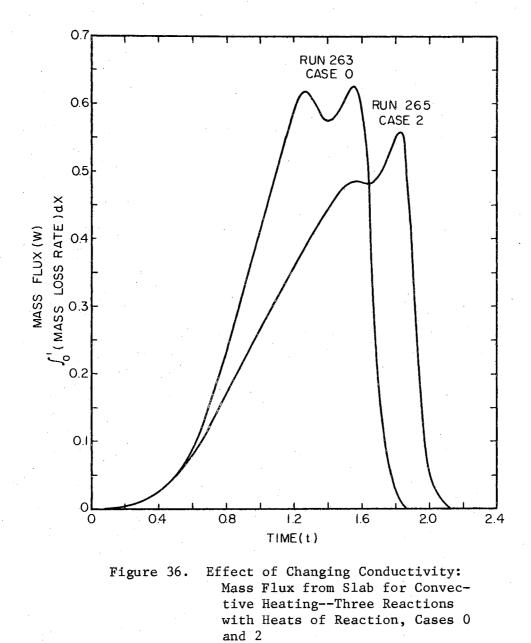


Figure 35. Effect of Variable Conductivity and Heat Transfer Coefficients: Temperature History of Front and Back Surfaces for Convective Heating--Three Reactions with Heats of Reaction, Cases 0 and 2

contribution from the conductivity term. Figure 36 shows that the mass flux is modified from the resulting lower surface heat flux.

This section presented the results of three competitive and consecutive reactions with endothermic and exothermic heats of reaction. It was found that a small change in the heat flux parameter greatly affects the temperature history, mass flux, and final char distributions. Higher heat flux values also make heat transfer coefficients and effective thermal conductivity effects more important. A major effect of the competing reaction is to cause a change in the character of the reactions throughout the slab.



CHAPTER V

SUMMARY

The literature pertaining to the pyrolysis of porous solids was reviewed for information regarding the extent of previous mathematical analyses and experimental determination of the important physical events. As a result, the mathematical analysis was extended to include competitive and consecutive, homogeneous reactions with reactant depletion and consideration of both a constant and density-dependent, effective thermal conductivity. Two types of surface heating for a finite thickness, infinite slab were considered: a constant heat flux, radiative type, and a constant source temperature, convective type. A densitydependent, as well as a constant, heat transfer coefficient for convective heating was included in the various cases that were studied. The characteristic non-dimensional parameters were also developed for both types of boundary conditions.

In order to solve the resulting unsteady, non-linear, second-order, partial differential equation, a numerical technique was developed, based on the method of Integral Relations. The technique requires only a small amount of computer storage and can obtain accurate results for most cases with three to six approximating strips. The ability to obtain better approximations by increasing the number of approximating strips is limited only by the longer computer times required.

The computed results presented temperature and species histories

and slab profiles as well as mass flux, reaction rates, and weight loss information. It was shown that the heat generated or absorbed from the reactions has a great effect on the solid ahead of and behind the reactions as well as in the reaction zone, not only with respect to temperature, but also reaction rates and mass flux. An increasing surface heat flux increases the maximum reaction rate and the temperature at which it occurs.

The effect of a density-dependent, effective thermal conductivity and heat transfer coefficient is negligible for convective heating when source temperatures are low. For radiant heating and high source temperature convective heating, the change in effective thermal conductivity can greatly affect the temperature history of the slab. This is especially true as the final densities become smaller. These effects must be carefully considered when interpreting temperature histories.

Small changes in the activation energies, frequency factors, and surface heat flux, greatly affect the time and rate of production of species for competing reactions while only slightly affecting the quantity produced.

The competing reactions cause a change in the character of the overall reaction throughout the slab and enable small changes in the surface heat flux to greatly affect the temperature history, mass flux, and final char distributions.

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APPENDIX A

INTEGRATION OF EQUATION (3.1)

Starting with Equation (3.1), each term can be integrated as shown.

$$\int_{0}^{1} f_{m}(x) \frac{\partial T}{\partial t} = \int_{0}^{1} f_{m}(x) \frac{\partial^{2}T}{\partial x^{2}} dx + \int_{0}^{1} f_{m}(x) \frac{1}{\rho} \frac{\partial \rho}{\partial x} \frac{\partial T}{\partial x} dx$$

$$I \qquad II \qquad III \qquad III$$

$$- \frac{1}{G} \sum_{i=1}^{3} \int_{0}^{1} f_{m}(x) \overline{H}_{i} dx \qquad (3.1)$$

IV

Term I:

Using Leibnitz's rule

$$\frac{d}{dt} \int_0^1 f_m(x) T dx = \int_0^1 f_m(x) \frac{\partial T}{\partial t} dx$$
I

Term II:

Integrating by parts

$$\int_{0}^{1} f_{m}(\mathbf{x}) \frac{\partial^{2} T}{\partial \mathbf{x}^{2}} d\mathbf{x} = f_{m} \frac{\partial T}{\partial \mathbf{x}} \Big|_{0}^{1} - \frac{df_{m}}{d\mathbf{x}} T \Big|_{0}^{1} + \int_{0}^{1} \frac{d^{2} f_{m}}{d\mathbf{x}^{2}} T d\mathbf{x}$$
(A.1)
II

Since

$$f_{m}(x) = 1, x, x^{2}, ..., x^{(N-1)} = x^{(m-1)}, (m = 1, 2, ..., N)$$
(A.2a)

$$\frac{df_{m}}{dx} = (m - 1) x^{(m-2)}, \frac{df_{1}}{dx} = 0, \frac{df_{2}}{dx} = 1$$
 (A.2b)

and

$$\frac{d^2 f_m}{dx^2} = (m - 1)(m - 2) x^{(m-3)}$$
(A.2c)

Therefore, Equation (A.1) is

$$\int_{0}^{1} f_{m} \frac{\partial^{2}T}{\partial x^{2}} dx = \frac{\partial T}{\partial x}\Big|_{x=1}^{2} - \frac{\partial T}{\partial x}\Big|_{x=0}^{2} - (m-1)T\Big|_{x=1}^{2} + \begin{cases} T \\ 0 \end{cases} x = 0; m = 2 \\ m \neq 2 \end{cases}$$
III
$$+ (m-1)(m-2) \int_{0}^{1} x^{(m-3)} T dx.$$

Term III:

Write $\frac{1}{\rho} \frac{\partial \rho}{\partial \mathbf{x}}$ as $\frac{\partial (\ln \rho)}{\partial \mathbf{x}}$ then, integrate Term III by parts.

$$\int_{0}^{1} f_{m} \frac{\partial (\ln \rho)}{\partial x} \frac{\partial T}{\partial x} dx = f_{m} \cdot \ln \rho \cdot \frac{\partial T}{\partial x} \Big|_{0}^{1} - \int_{0}^{1} f_{m} \cdot \ln \rho \cdot \frac{\partial^{2} T}{\partial x^{2}} dx$$

$$III$$

$$- \int_{0}^{1} \ln \rho \cdot \frac{df_{m}}{dx} \cdot \frac{\partial T}{\partial x} dx$$

Using Equation (A.2)

$$\int_{0}^{1} f_{m} \frac{\partial (\ln \rho)}{\partial x} \frac{\partial T}{\partial x} dx = \ln \rho \cdot \frac{\partial T}{\partial x} \Big|_{x=1} - \int_{0}^{1} x^{(m-1)} \ln \rho \cdot \frac{\partial^{2} T}{\partial x^{2}} dx$$
III
$$- (m-1) \int_{0}^{1} \ln \rho \cdot x^{(m-2)} \cdot \frac{\partial T}{\partial x} dx$$

Term IV:

Substitute for $f_m(x)$ from Equation (A.2a)

$$\frac{1}{G}\sum_{i=1}^{3}\int_{0}^{1}f_{m}\overline{H}_{i} dx = \frac{1}{G}\sum_{i=1}^{3}\int_{0}^{1}x^{(m-1)}\overline{H}_{i} dx$$

Putting the equivalent expression for each term back into Equation (3.1) yields Equation (3.4).

APPENDIX B

INTEGRATION OF EQUATION (3.4)

Substituting the approximations (3.5) into Equation (3.4) and multiplying the x-polynomials result in

$$\frac{d}{dt} \int_{0}^{1} \sum_{n=1}^{N+2} A_{(n)} x^{(m+n-2)} dx = \frac{\partial T}{\partial x} \Big|_{x=1} - (m-1) \sum_{n=1}^{N+2} A_{(n)} + A_{(1)} \Big|_{m=2} + (m-1)(m-2) \int_{0}^{1} \sum_{n=1}^{N+2} A_{(n)} x^{(m+n-4)} dx + \ln \rho \cdot \frac{\partial T}{\partial x} \Big|_{x=1} - \int_{0}^{1} \sum_{n=1}^{N} A_{(n)} x^{(m+n-2)} dx - (m-1) \int_{0}^{1} \sum_{n=1}^{N} J_{1}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N} J_{2}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{2} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{N} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{N} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N+1} H_{i}_{(n)} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{N} \frac{R_{i}}{F_{i}} \int_{0}^{1} \sum_{n=1}^{N} \frac{R_{i}}{F_{i}} x^{(m+n-2)} dx - \frac{1}{G} \sum_{i=1}^{N} \frac{R_{i}}{F_{i}} x^{(m+n-2)} dx -$$

Integrate Equation (B.1) and obtain

$$\sum_{n=1}^{N+2} \left(\frac{1}{m+n-1}\right) \frac{dA_{(n)}}{dt} = \left[\left(1 + \ln \rho\right) B\right]_{x=1} - (m-1) \sum_{n=1}^{N+2} A_{(n)} + A_{(1)}\right]_{m=2} + (m-1)(m-2) \sum_{n=1}^{N+2} \left(\frac{1}{m+n-3}\right) A_{(n)} - \sum_{n=1}^{N} \left(\frac{1}{m+n-1}\right) J_{1(n)} - (m-1) \sum_{n=1}^{N} \left(\frac{1}{m+n-2}\right) J_{2(n)} - \frac{1}{G} \sum_{i=1}^{3} \frac{R_{i}}{F_{i}} \sum_{n=1}^{N+1} \left(\frac{1}{m+n-1}\right) H_{i(n)}$$
(B.2)

Where $B = \frac{\partial T}{\partial x} \Big|_{x=1}$ and depends on the type and case of boundary condition

being considered. Terms 2, 3, and 4 to the right of the equal sign in Equation (B.2) can be combined in the following way: First, rewrite as

$$-(m-1)A_{(1)} - (m-1)\sum_{n=2}^{N+2}A_{(n)} + A_{(1)}\Big|_{m=2} + (m-1)A_{(1)}\Big|_{m\geq 3}$$

+
$$(m - 1)(m - 2) \sum_{n=2}^{N+2} (\frac{1}{m + n - 3}) A_{(n)}$$

then, the $A_{(1)}$ terms will cancel for all values of m and the remaining terms will combine to give

$$\sum_{n=2}^{N+2} \left[\frac{(m-1)(m-2)}{(m+n-3)} - (m-1) \right] A_{(n)} = -(m-1) \sum_{n=2}^{N+2} \frac{(n-1)}{(m+n-3)} A_{(n)}$$

When the temperature approximation (3.5a) is used in the boundary condition (2.11d), the result is that $A_{(2)}(t) = 0$. Therefore, the summation index on the above term can be changed to

$$- (m - 1) \sum_{n=1}^{N} \frac{(n + 1)}{(m + n - 1)} A_{(n+2)}$$
(B.3)

Likewise, the first term of Equation (B.2) can be written as

$$\frac{1}{m} \frac{dA_{(1)}}{dt} + \sum_{n=1}^{N} \left(\frac{1}{m+n+1} \right) \frac{dA_{(n+2)}}{dt}$$
(B.4)

Substituting expressions (B.3) and (B.4) into Equation (B.2) gives Equation (3.6).

APPENDIX C

EVALUATION OF INITIAL TEMPERATURE COEFFICIENTS

The temperature approximation (3.5a) is used in the boundary conditions (2.11a, d) to evaluate the initial values for the coefficients in the following way.

Since

T(x, 0) = 0

then

$$\sum_{n=1}^{N+2} A_{(n)}(0) x^{(n-1)} = 0$$
 (C.1)

giving

$$A_{(1)}(0) = 0 . (C.2)$$

Since

$$\frac{\partial T}{\partial x}\Big|_{x=0} = 0$$

then

$$A_{(2)}(0) = 0 (C.3)$$

At x = 1, t = 0;
$$\frac{\partial T}{\partial x} = \frac{q(0)}{\rho(1, 0)} = \frac{1}{1}$$
, all cases

Therefore,

$$\sum_{n=3}^{N+2} (n - 1) A_{(n)}(0) = 1$$
 (C.4)

Equations (C.1) to (C.4) are also Equations (3.9).

APPENDIX D

CALCULATION OF THE TEMPERATURE IN

AN INERT SLAB BY THE

INTEGRAL METHOD

For the inert slab only the first two terms of Equation (2.9) are needed:

$$\frac{\partial \mathbf{T}}{\partial \mathbf{t}} = \frac{\partial^2 \mathbf{T}}{\partial \mathbf{x}^2} \tag{D.1}$$

The boundary conditions will be Equation (2.11a, d, f):

$$T(x, 0) = 0$$
 (D.2)

$$\frac{\partial \mathbf{T}}{\partial \mathbf{x}}\Big|_{\mathbf{x}=\mathbf{0}} = \mathbf{0} \tag{D.3}$$

$$\frac{\partial T}{\partial x}\Big|_{x=1} = 1$$
 (constant heat flux) (D.4)

Following the steps outlined in Chapter III, "Summary of the Integral Technique," the slab is divided by two planes (N = 2), the surfaces x = 0 and x = 1. Equation (D.1) is multiplied by the weighting functions, $f_m(x) = 1$, x, and integrated over the slab:

$$\frac{d}{dt} \int_0^1 1 \cdot T \, dx = 1 \cdot \frac{\partial T}{\partial x} \Big|_{x=1} - 1 \cdot \frac{\partial T}{\partial x} \Big|_{x=0} - T(1, t) \cdot 0 + T(0, t) \cdot 0$$
(D.5a)

$$\frac{\mathrm{d}}{\mathrm{d}t} \int_{0}^{1} \mathbf{x} \cdot \mathbf{T} \, \mathrm{d}\mathbf{x} = (1) \cdot \frac{\partial \mathbf{T}}{\partial \mathbf{x}} \Big|_{\mathbf{x}=1} - (0) \left. \frac{\partial \mathbf{T}}{\partial \mathbf{x}} \right|_{\mathbf{x}=0} - \mathbf{T}(1, t) \cdot 1 + \mathbf{T}(0, t) \cdot 1$$
(D.5b)

Using the temperature approximation, Equation (3.5a),

$$T(x, t) = \sum_{n=1}^{N+2} A_{(n)}(t) x^{(n-1)} = A_{(1)} + A_{(2)}x + A_{(3)}x^{2} + A_{(4)}x^{3}$$
(D.6)

Equation (D.3) gives

$$A_{(2)} = 0$$
 (D.7)

while Equation (D.4) gives

$$2A_{(3)} + 3A_{(4)} = 1$$
 (D.8)

Using Equation (D.6) and (D.7) in (D.5) gives

$$\frac{d}{dt} \int_0^1 (A_{(1)} + A_{(3)}x^2 + A_{(4)}x^3) dx = 1$$
 (D.9a)

$$\frac{d}{dt} \int_{0}^{1} (A_{(1)}x + A_{(3)}x^{3} + A_{(4)}x^{4}) dx = 1 - A_{(3)} - A_{(4)}$$
(D.9b)

Integrating

$$\frac{dA_{(1)}}{dt} + \frac{1}{3} \frac{dA_{(3)}}{dt} + \frac{1}{4} \frac{dA_{(4)}}{dt} = 1$$
 (D.10a)

$$\frac{1}{2} \frac{dA(1)}{dt} + \frac{1}{4} \frac{dA(3)}{dt} + \frac{1}{5} \frac{dA(4)}{dt} = 1 - A_{(3)} - A_{(4)} \quad (D_{\circ}10b)$$

Taking the derivative of Equation (D.8) and using Equation (D.10) eliminates $\frac{dA_{(4)}}{dt}$:

$$\frac{dA_{(4)}}{dt} = \frac{1}{3} - \frac{2}{3} \frac{dA_{(3)}}{dt}$$
$$\frac{dA_{(1)}}{dt} + \frac{1}{6} \frac{dA_{(3)}}{dt} = 1$$
(D.11a)

$$3 \frac{dA(1)}{dt} + \frac{7}{10} \frac{dA(3)}{dt} = 4 - 2A_{(3)}$$
(D.11b)

Solving Equation (D.11) simultaneously gives explicit equations for each coefficient:

. .

$$\frac{dA_{(1)}}{dt} = \frac{1}{6} + \frac{5}{3} A_{(3)}$$
(D.12a)

$$\frac{dA_{(3)}}{dt} = 5 - 10A_{(3)}$$
(D.12b)

Integrating (D.12b)

$$A_{(3)} = \frac{1}{2} + c_1 e^{-10t}$$
 (D.13)

Substituting Equation (D.13) into (D.12a) and integrating

$$A_{(1)} = t - \frac{1}{6} c_1 e^{-10t} + c_2$$
 (D.14)

Using the initial conditions (D.2) at both N strips together with Equations (D.8), (D.13), and (D.14) evaluates the two constants. The coefficients then become

$$A_{(1)} = t + \frac{1}{4} (e^{-10t} - 1)$$

$$A_{(3)} = \frac{1}{2} (1 - 3e^{-10t})$$

$$A_{(4)} = e^{-10t}$$

$$A_{(2)} = 0$$

Equation (D.6) then becomes

$$T(x, t) = t + \frac{1}{2}x^2 - \frac{1}{4} + \frac{1}{4}(1 - 6x^2 + 4x^3)e^{-10t}$$

Compare with the exact solution from Carslaw and Jaeger (6):

T(x, t) = t +
$$\frac{1}{2}$$
 x² - $\frac{1}{6}$ + $\frac{2}{\pi^2}$ (1 - $\frac{\pi^2}{2}$ x² + $\frac{\pi^4}{24}$ x⁴ - . . .) e^{- π^2 t} - . . .

APPENDIX E

CALCULATION OF GAS SPECIES

By identifying the fraction of each solid species that is consumed or produced in the various reactions, the mass balance equations (2.5) can be used to calculate the gas species.

 $\rho_{1,1}$ = mass of solid₁ converted to product in reaction 1 $\rho_{1,2}$ = mass of solid₁ converted to product in reaction 2 $\rho_{2,1}$ = mass of solid₂ produced in reaction 1 $\rho_{2,3}$ = mass of solid₂ converted to product in reaction 3

 ρ_1 , ρ_2 , and ρ_6 are the calculated solid species that are present at a given instant. ρ_3 , ρ_4 , and ρ_5 are the gas species that have been produced up to a given point in time.

Then,

$$1 = \rho_1 + \rho_{1,1} + \rho_{1,2}$$

 $c_1\rho_{1,1} = \rho_{2,1} = \rho_2 + \rho_{2,3}$; $(1 - c_1) \rho_{1,1} = \rho_3$

$$\rho_{1,2} = \rho_4$$

$$c_2\rho_2 = \rho_6$$
; $(1 - c_2)\rho_2 = \rho_5$

Using these relations, the amount of each gas species can be found from the solid species:

$$\rho_{2,1} = \rho_{2} + \frac{\rho_{6}}{c_{2}}; \quad \rho_{1,1} = \frac{\rho_{2}}{c_{1}} + \frac{\rho_{6}}{c_{1}c_{2}}$$

$$\rho_{3} = (1 - c_{1})(\frac{\rho_{2}}{c_{1}} + \frac{\rho_{6}}{c_{1}c_{2}}) = Gas_{3} \quad (E.1a)$$

$$\rho_4 = 1 - \rho_1 - \left(\frac{\rho_2}{c_1} + \frac{\rho_6}{c_1c_2}\right) = Gas_4$$
 (E.1b)

$$\rho_5 = \frac{(1 - c_2)}{c_2} \rho_6 = Gas_5$$
 (E.1c)

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Equations (E.1) then become Equations (3.11).

APPENDIX F

DERIVATION OF EQUATION (2.4e)

Start with the species equation for $Solid_1$ (2.8a)

$$\frac{\partial \rho_1^*}{\partial t^*} = -\rho_1^*(k_1 + k_2) ,$$

divide by ρ_1^{\bigstar} ,

$$\frac{\partial(\ln\rho^{\dagger})}{\partial t^{\star}} = -(k_1 + k_2) ,$$

and take the x-derivative

$$\frac{\partial}{\partial \mathbf{x}^{\star}} \left(\frac{\partial \ln \rho_1^{\star}}{\partial t^{\star}} \right) = -\left(\frac{\partial k_1}{\partial \mathbf{x}^{\star}} + \frac{\partial k_2}{\partial \mathbf{x}^{\star}} \right) . \tag{F.1}$$

Since $k_i = a_i \exp[(-E_i^*)/(R^{\circ}T^*)]$, then

$$\frac{\partial \mathbf{k}_{\mathbf{i}}}{\partial \mathbf{x}^{*}} = \frac{\mathbf{a}_{\mathbf{i}} \mathbf{E}^{*}_{\mathbf{i}}}{\mathbf{R}^{\circ} \mathbf{T}^{*2}} \frac{\partial \mathbf{T}^{*}}{\partial \mathbf{x}^{*}} \exp(\frac{-\mathbf{E}^{*}_{\mathbf{i}}}{\mathbf{R}^{\circ} \mathbf{T}^{*}})$$

at x = 0, Equation (2.4d) gives

$$\frac{\partial T^*}{\partial x^*} = 0$$

thus, at x = 0

$$\frac{\partial k_{i}}{\partial x^{*}} = 0$$

Equation (F.1) then becomes

$$\frac{\partial}{\partial \mathbf{x^*}} \left(\frac{\partial \ln \rho_1^*}{\partial t^*} \right) = 0$$

Reverse the order of differentiation,

$$\frac{\partial}{\partial t^{\star}} \left(\frac{\partial \ln \rho_1^{\star}}{\partial x^{\star}} \right) = 0$$

integrating,

$$\frac{\partial \ln \rho_1^*}{\partial x^*} = f(x) + c_1$$

or also,

$$\frac{1}{\rho_1^*} \frac{\partial \rho_1^*}{\partial \mathbf{x}^*} = \mathbf{f}(\mathbf{x}) + \mathbf{c}_1$$

Since $\frac{1}{\rho_1^*} \frac{\partial \rho_1^*}{\partial x^*}$ is not a function of time, it can be determined by evaluating it at any one value of time. At t = 0, $\rho_1^* = \rho_0^*$, a constant. Therefore, $\frac{\partial \rho_1^*}{\partial x^*} = 0$ for all time at x = 0. The same approach is taken for other solid species so that Equation (2.4e) can be shown for the total solid by

$$\frac{\partial \rho^{\star}}{\partial \mathbf{x}^{\star}} = \frac{\partial \rho_{1}^{\star}}{\partial \mathbf{x}^{\star}} + \frac{\partial \rho_{2}^{\star}}{\partial \mathbf{x}^{\star}} + \frac{\partial \rho_{6}^{\star}}{\partial \mathbf{x}^{\star}} = 0$$

at x = 0 for all time.

APPENDIX G

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LISTING OF COMPUTER PROGRAM VARIABLES

Main Program

Computer Variable with	Mathematical					
Maximum Dimension	Variable	Description				
A(N+2)	A(n)	Polynomial temperature coefficient				
B(N*N)		Matrix coefficients for columnwise input to SIMQdestroyed in compu- tation				
E(3)	E i	Activation energy parameter for reaction i				
G	G	Surface heat flux parameter				
Ν	Ň	Number of interpolation strips				
R(3)	R _i	Heat of reaction parameter				
Т	t	Time variable				
X(N)	x j	x-value of j th interpolation strip				
Y(202)		Variables used in START and KAMSUB to store unknowns and derivatives				
AB(N)		Vector of original constants for input to SIMQReturns solution values				
BO	Nu	Nusselt number				
CS(N+1)		Polynomial coefficients for average solid (weight)				
C1(N)	^ρ 1,j	Local density of $Solid_1$				
C2(N)	^ρ 2,j	Local density of Solid ₂				

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Computer Variable with Maximum Dimension	Mathematical Variable	Description				
C3(N)	^ρ 3,j	Local total mass of Gas ₃ produced from strip j				
C4(N)	^ρ 4,j	Local total mass of Gas_4 produced from strip j				
C5(N)	^ρ 5,j	Local total mass of Gas_5 produced from strip j				
C6(N)	^p 6,j	Local density of Solid ₆				
DI	n	Summation dummy index				
Fl	F ₁	Frequency factor parameter for reaction 1				
F2	F ₂	Frequency factor parameter for reaction 2				
F3	F ₃	Frequency factor parameter for reaction 3				
KS		<pre>Flag in SIMQ: 0 = normal solution, 1 = singular solution (see Appen- dix H)</pre>				
LN	(N-1)	Program constant				
NE	(4N+2)	Number of unknownsfor START and KAMSUB				
S3(N+1)		Polynomial coefficients for total mass of Gas_3 produced from slab				
S4(N+1)		Polynomial coefficients for total mass of Gas ₄ produced from slab				
S5(N+1)		Polynomial coefficients for total mass of Gas_5 produced from slab				
XJ		Intermediate computational value				
XP		Intermediate computational value				
AP1	(N+1)	Program constant				
AP2	(n+2)	Summation dummy index + 2				
BCS($(N-1)^2$)		Matrix coefficients for total solid polynomialinput to SIMQ				

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Computer Variable with	Mathematical	
Maximum Dimension	Variable	Description
$BDC((N-1)^2)$		Matrix coefficients for gas gener- ation polynomialinput to SIMQ
BS3((N-1) ²)		Matrix coefficients for total Gas ₃ polynomialinput to SIMQ
BS4((N-1) ²)		Matrix coefficients for total Gas ₄ polynomialinput to SIMQ
BS5((N-1) ²)		Matrix coefficients for total Gas ₅ polynomialinput to SIMQ
CS1	cl	Mass fraction of Solid $_1$ converted to Solid $_2$
CS2	c ₂	Mass fraction of Solid $_2$ converted to Solid $_6$
DCT(N+1)		Polynomial coefficients for gas generation rate
DEL		Initial time step-size
DLP		Fraction used to reduce time step- size
DSM(N)		Local solid density time-derivative
IBC		Boundary condition type: 0 = Type I, 1 = Type II (see Chapter II)
IM1	(n-1)	Summation dummy index - 1
IP1	(n+1)	Summation dummy index + 1
IP2	(n+2)	Summation dummy index + 2
JM1	(m-1)	Summation dummy index - 1
MER		Error calculation option in KAMSUB: 1 = relative error, 2 = absolute error
МОР		Integration option for KAMSUB: 1 = Runga-Kutta, 2 = Adams fixed step size, 3 = Adams variable step size
NP1	(N+1)	Program constant

Computer Variable with Maximum Dimension	Mathematical Variable	Description
NP2	(N+2)	Program constant
NSQ	N ²	Program constant
RCS(N)	ρj	Local total solid density at strip j
WGT	· .	Slab weightaverage solid density
XJI		Intermediate computational value
ASUM		Intermediate computational value
BBCS(N-1)		Vector of original constants for input to SIMQ
BBDC(N-1)		Vector of original constants for input to SIMQ
BBS3(N-1)		Vector of original constants for input to SIMQ
BBS4(N-1)		Vector of original constants for input to SIMQ
BBS5(N-1)		Vector of original constants for input to SIMQ
CSUM		Intermediate computational value
DELT		Difference in present and previous- ly printed time-step
DLMN		Minimum allowable time-step in KAMSUB
DLMX		Maximum allowable time-step in KAMSUB
ERMN		Minimum allowable error for KAMSUB
ERMX		Maximum allowable error for KAMSUB
FLUX		Mass flux from slab
FSUM		Intermediate computational value
SPC3		Total mass of Gas ₃ produced from slab

Computer Variable with <u>Maximum Dimension</u>	Mathematical Variable	Description			
SPC4		Total mass of Gas ₄ produced from slab			
SPC5		Total mass of Gas ₅ produced from slab			
SSFX (N)	· .	Local gas generation rate			
TEND		Time value to end calculation			
TMIN		Minimum value of DELT for printing all values			
TPRT		Value of time when values are printed			
XPTH		Intermediate computational value			
IÇOND		Effective thermal conductivity case: 0 = Case 0, 1 = Case 1, 2 = Case 2 (see Chapter II)			
NPRNT		Value of i for printing every i th time step			
TMPTR(N)		Local temperature			
MCOUNT		<pre>Punch and/or print option: 0 = print only, 1 = print and punch</pre>			
NCOUNT		Number of time-steps computed since the last printed step			

Subroutine DERFUN

Variables already described in the Main Program will not be listed as they are either in common or perform the same function.

Computer Variable with Maximum Dimension	Mathematical Variable	Description
М	m	Summation dummy index
Q(3)		Intermediate computational value

Computer Variable with Maximum Dimension	Mathematical Variable	Description					
AN	N	Number of interpolation strips					
BG((N-1) ²)		Matrix coefficients for input to SIMQ					
BI	(n+1)	Summation dummy index + 1					
BN	(m+n-1)	Summation dummy indices - 1					
BT((N-1) ²)		Matrix coefficients for input to SIMQ					
DA(N+2)	$\frac{dA(n)}{dt}$	Derivative of polynomial tempera- ture coefficients					
GS(N)	J _{1(n)} (t)	Polynomial coefficients for effec- tive conductivity integral approxi- mation					
GT(N)	J _{2(n)} (t)	Polynomial coefficients for effec- tive conductivity integral approxi- mation					
HS(3, N+1)	H _{i,(n)} (t)	Polynomial coefficients for heat generation from reaction i integral approximation					
QS	q(ţ)	Time dependent surface heat flux function					
BBG(N-1)		Vector of original constants for input to SIMQ					
BBT(N-1)		Vector of original constants for input to \$IMQ					
BH1((N+1) ²)		Matrix coefficients for input to SIMQ					
BH2((N-1) ²)		Matrix coefficients for input to SIMQ					
BH3((N-1) ²)		Matrix coefficients for input to SIMQ					
BII	n(n+1)	Summation dummy index term					
BMN	(m+n+1)	Summation dummy indices + 1					
BM1	m	Summation dummy index					

Computer Variable with <u>Maximum Dimension</u>	Mathematical Variable	Description
BM2	(m-1)	Summation dummy index - 1
BM3	(n+1)	Summation dummy index + 1
CK1	. 1	Intermediate computational value
CK2		Intermediate computational value
CK3		Intermediate computational value
DC1(N)	$\frac{dC_{1(n)}}{dt}$	Local time derivative of Solid $_1$ density
DC2(N)	$\frac{dC_{2(n)}}{dt}$	Local time derivative of Solid ₂ density
DC6(N)	$\frac{dC_{6(n)}}{dt}$	Local time derivative of Solid ₆ density
DQS	<u>dq</u> dt	Derivative of time dependent sur- face heat flux function
DRG		Intermediate computational value
TP1		Intermediate computational value
TP2	-	Intermediate computational value
TP 3		Intermediate computational value
XX1	(m+1)	Summation dummy index
ANP 2		Intermediate computational value
ASRF		Intermediate computational value
BBH1(N+1)		Vector of original constants for input to SIMQ
BBH2(N+1)	•	Vector of original constants for input to SIMQ
BBH3(N+1)		Vector of original constants for input to SIMQ
BRCS	,	Intermediate computational value
BSUM		Intermediate computational value
CNDX		Intermediate computational value

Computer Variable						
with <u>Maximum Dimension</u>	Mathematical Variable	Description				
DNDX		Intermediate computational value				
DSUM		Intermediate computational value				
SUBK(3, N)	ī,(n)	Exponential term of local reaction rate constant for reaction i				
TSLP		Intermediate computational value				
XJII		Intermediate computational value				
XJP1		Intermediate computational value				

APPENDIX H

DESCRIPTION OF THE SUBROUTINES SIMQ, START, AND KAMSUB

Description of the Subroutine $\ensuremath{\mathsf{SIMQ}}$

	SIMO
0 3 8 9 9 8 9 9 9 9 9 9 9 9 9 9 9 9 9 9 9	• SIMQ
FURBOUTINE STWO	SIMQ
	SIMQ SIMQ
	SIMO
	SIMQ
	SIMQ
	SIMQ
USAGE	SIMQ
CALL SIMQ(A0BONOKS)	SIMQ
	SIMO
DESCRIPTION OF PARAMETERS	SIMQ
	SIMQ
	SIMO
	SIMQ
	SIMQ
	SIMO
	SIMO
	SIMO
REMARKS	SIMO
MATRIX A MUST BE GENERAL.	SIMO
IF MATRIX IS SINGULAR > SOLUTION VALUES ARE MEANINGLESS.	SIMQ
AN ALTERNATIVE SOLUTION MAY BE OBTAINED BY USING MATRIX	SIMQ
INVERSION (MINV) AND MATRIX PRODUCT (GMPRD).	SIMQ
· · · · · · · · · · · · · · · · · · ·	SIMQ
	SIMQ
NONE	SIMQ
	SIMQ
	SIMQ
	SIMQ
	SIMO
	SIMQ
	SIMQ
	SIMO
	SIMO
VALUES ARE DEVELOPED IN VECTOR B: WITH VARIABLE 1 IN B(1):	
VARIABLE 2 IN B(2), VARIABLE N IN B(N).	SIMO
IF NO PIVOT CAN BE FOUND EXCEEDING A TOLERANCE OF 0.0,	SIMQ
THE MATRIX IS CONSIDERED SINGULAR AND KS IS SET TO 1. THIS	SIMQ
TOLERANCE CAN BE MODIFIED BY REPLACING THE FIRST STATEMENT.	SIMQ
	SIMQ
	<pre>SUBROUTINE SIMQ PURPOSE OBTAIN SOLUTION OF A SET OF SIMULTANEOUS LINEAR EQUATIONS, AX=B USAGE CALL SIMQ(A.B.N.KS). DESCRIPTION OF PARAMETERS A - MATRIX OF COEFFICIENTS STORED COLUMNWISE. THESE ARE DESTROYED IN THE COMPUTATION. THE SIZE OF MATRIX A IS N BY N. B - VFCTOR OF ORIGINAL CONSTANTS (LENGTH N). THESE ARE REPLACED BY FINAL SOLUTION VALUES, VECTOR X. N - NUMBER OF EQUATIONS AND VARIABLES. N MUST BE .GT. ONE. KS - OUTPUT DIGIT C FOR A NORMAL SOLUTION VALUES ARE MEANINGLESS. AN ALTERNATIVE SOLUTION MAY BE OBTAINED BY USING MATRIX INVERSION (MINV) AND MATRIX PRODUCT (GMPRD). SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED NONE METHOD METHOD OF SOLUTION IS BY ELIMINATION USING LARGEST PIVOTAL DIVISOR. EACH STAGE OF ELIMINATION CONSISTS OF INTERCHANGING REMARD SOLUTION TO OBTAIN VARIABLE N IS DONE IN N STAGES. THE BACK SOLUTION FOR THE OTHER VARIABLES IS CALCULATED BY SUCCESSIVE SUBSTITUTIONS, FINAL SOLUTION VARIABLE 2 IN B(2)</pre>

Description of the Subroutines START and KAMSUB

Solution of Ordinary Differential Equations

<u>Purpose</u>. Numerical solution of Ne simultaneous first-order ordinary differential equations.

Type. FORTRAN IV.

<u>Mode of Operation</u>. Real, single, and double precision arithmetic. <u>General Description</u>. The available subroutine package performs numerical integration of Ne (Ne ≤ 100) simultaneous first-order ordinary differential equations. The integration may be carried forward or backward from a specified Ne-set of one-point boundary conditions. The user has the option of selecting the Runge-Kutta or the Adams-Moulton method of numerical integration. When using the Adams-Moulton technique, the user may try a fixed step size or he may let the routine calculate its own step size based on minimum and maximum limits on the integration errors.

In addition to the user's main or calling program, three subroutines are used. Two of these are SUBROUTINE KAMSUB (NSTART) and SUB-ROUTINE START (M1, M2, M3, A1, A2, A3, A4, A5, A6). The third, entitled SUBROUTINE DERFUN, is prepared by the user to define the functional forms of his differential equations.

The internal operation of the integration is carried out in doubleprecision arithmetic in order to improve the accuracy. The solutions available externally are in single precision numbers.

Instructions for Usage.

 The user provides in his main program the following sequence of instructions:

- a) Declare a common block: COMMON Y(202)
- b) Define the boundary conditions:

Y(1), Y(2), . . . , Y(Ne), Y(Ne + 1)

where Y(Ne + 1) is the initial value of the independent variable.

c) Call the subroutine START to begin the integration process. The subroutine requires that nine arguments be provided. The function of each of these parameters is described below.

CALL START (M1, M2, M3, A1, A2, A3, A4, A5, A6) where M1, M2, M3 are fixed-point numbers and A1, A2, . . ., A6 are floating-point numbers.

M1 - The number of equations (M1 \leq 100)

M2 - Option Parameter

M2 = 1 Integrates by Runge-Kutta Method

- M2 = 2 Integrates by Adams-Moulton Method with fixed step size
- M2 = 3 Integrates by Adams-Moulton Method with the step size internally calcu-

lated based on specified error limits.

M3 - Option parameter for error determination.

M3 = 1 Error calculated as relative error.

M3 = 2 Error calculated as absolute error.

Al - The step size. (If M2 = 3, this is used as a first trial value and then adjusted so that the error conforms to the limits.) A negative value causes backward integration.

A2 - Maximum value of integration error at each step.

122

A4 - Largest permitted step size.

A5 - Smallest permitted step size.

A6 - The factor used to reduce the step size when M2 = 3and the integration error is too large.

Notes: (1) M3, A2 through A6 are applicable only if M2 = 3.

(2) If A3 = 0, the routine sets A3 = A2/55.

- (3) A6 is specified normally as 0 <A6 < 1; however, an initial value of zero will cause the routine to set A6 = 1/2.
- d) Call the subroutine KAMSUB (1) for each step of the independent variable. The call is repeated until the integration has been carried to the desired final value of the independent variable. To preserve the solution for later calculations or printing, it is necessary to store the values after each call of KAMSUB. The block Y(202) only holds the current values of the solution.
- 2) The user defines the specific functional forms of the differential equation to be integrated in a subroutine DERFUN. The equations must be expressed as first-order equations. The common block Y is utilized as follows:

Y(1), Y(2), . . , Y(Ne) - The values of the dependent variables.

Y(Ne + 1) - The corresponding value of the independent variable.

Y(Ne + 2) - The current step size.

Y(Ne + 3), . . , Y(2Ne + 2) - The current values of the

derivatives.

where Y(Ne + 3) is the derivative of Y(1) and Y(Ne + 4) is the derivative of Y(2), etc.

Caution to User

A labeled common block SHARE is used to transfer data internally. The user should not use a common area with this name.

APPENDIX I

COMPUTER LISTING OF ALL PROGRAMS

80/80 LIST

0000000011111111222222222333333333334444444444
1234567590123456789012345
CARD
0001 [C + + + + + + + + + + + + + + + + + +
0002 C** MAIN program **
0033 C**IHIS PROGRAM IS THE ENTRY POINT TO THE INTEGRAL METHOD SOLUTION OF **
0004 C**IHE PARTIAL DIFFERENTIAL EQUATIONS. **
9005 C** **
0006 C**ALL ARRAYS ARE DIMENSIONED AND EQUIVALENCED ACCORDING TO THE ORDER **
0007_G**0E_THE_APPROXIMATION_AND_CAN_BE_INSERTED AS A GROUP BETWEEN THE **
0009 C*COMMON Y (202) AND THE FIRST FORMAT STATEMENT. **
ODIO C#*ALL INPUT DATA IS READ IN AT THE BEGINNING OF THE PROGRAM. **
0012 C**THE FUNCTION OF THIS BEGINNING ROUTINE IS TO CALCULATE THE INITIAL **
0013 C**VALUES FOR THE POLYNIMIAL TEMPERATURE COEFFICIENTS AND THE INITIAL **
0014 C**SOLID SPECIES VALUES, SEND THE NECESSARY INFORMATION TO THE ENTRY **
015 C**POINT OF THE OPDIMARY DIFFERENTIAL EQUATIONS INTEGRATION ROUTINE, **
0016 C****LCEIVE THE NEW INTEGRATED VALUES AT THE NEXT TIME STEP, MAKE **
0017 C**NECESSARY CALCULATIONS BASED ON THE NEW VALUES, PUNCH AND/OR PRINT **
OOIB C**THE DESIRED INFORMATION AND RETURN TO THE INTEGRATION FOR ANOTHER **
0019 C**TIME STEP. **
0020 C** ** **
0021 C##THE CALCULATION OF THE ORDINARY DIFFERENTIAL EQUATIONS BY THE ##
0022 C**INIFGRAL METHUD IS DONE IN ROUTINE 'DERFUN' WHICH IS CALLED ONLY **
0023 C##BY THE ORDINARY DIFF. EQ. ROUTINE.
0024 C** ** **
0025 C**THE ROUTINE 'SIMQ' SOLVES A SET OF SIMULTANEOUS ALGEBRAIC EQUATIONS**
0026 C**AND WILL PRINT AN S.S. MESSAGE IF THE MATRIX IS SINGULAR. **
0028 C 6+5 < < < < < < < < < < < < < < < < < < <
0029 COMMUN Y(202)
0030 DOUBLE PRECISION B(36), AB(6), x(6), G, BS3(25), BS4(25), BS5(25), BBS3(5
0031 1), BBS4(5), BBS5(5), DCT(7), BCS(25), BBCS(5), BDC(25), BBCC(5)
0032 CUMMON/CUMA/X,G,E(3),R(3),FLUX,LN,NSQ,N, NE,F1,F2,F3,CS1,CS2
0033 1,NP2,NP1,WGT, LCOND, IBC, BO, DSM(6), AP1
0034 DIMENSION C1(06), C2(06), C3(6), C5(6), C6(6), A(8), TMPTR(6), C4(6),
0035 1RCS(6), S3(7), S4(7), S5(7), S5FX(6), CS(7)
0036 EQUIVALENCE (Y(1),C1(1)),(Y(7),C2(1)),(Y(13),C6(1)),(Y(19),A(1)),
0037 1(Y(27),T)
0038 [000 FURMAT(13)
0039 1001 FURMAT(016.8)
0040 1010 FURMAT(5H X(J), 4X,L]HTEMPERATURE, 3X,L]HT0TAL SUL10, 4X,9HSPECIES
0041 1 1, 5X, 9HSPECIES 2, 5X, 9HSPECIES 6, 5X, 9HSPECIES 3, 5X, 9HSPECIES 4,
0042 25X, 9HSPECIES 5,15H MASS LOSS RATE)
0043 1111 FURMAT(3H J=,12,1X,9E14.5)
0044 1500 FORMAT(23H MASS FLUX AT SURFACE =, E16.6, 5X, 14HTOTAL WEIGHT =, E16.6
0046 2000 FORMAT(L16.8)
0047 2010 FURMAT(5H10ATA)
0048 2020 FURMAT(6H EI =,E16.6,6H E2 =,E16.6,6H E3 =,E16.6,6H R1 =,
0049 1E16.6.601 R2 = E16.6.6H R3 = E16.61
0950 2030 FURMAT(6H F1 =,E16.6,6H F2 =,E16.6,6H F3 =,E16.6,
0.051 1 6H $CS1=,F16.6, 6H$ $CS2=,F16.6$
0052 2040 FURMATIGH IEND=;E16.6.6H DEL=;E16.6.6H BD =;E16.6)
0053 2050 FORMATI6(2H X,12,2H =,D16.6))
0054 2055 FURMA1(5H G = ,D14.6)
0955 2070 FORMAT(3H N≈,13, 5X, 3HNE=,13, 5X,4HMOP=,13, 5X,4HMER=
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<pre>000000001111111111122222222223333333333</pre>
<pre>1,13, 5x,7HMCOUNT=,[3, 5x,6HNPRNT=,[3,5x,6HICOND=,13,5x,4HIBC=,[3] 22 FORMAT(11HOFOR TIME =,E13.5,5x,13HTOTAL GAS 3 =,E13.5,4x,13HTOTAL 1GAS 4 =,F13.5,4x,13HTOTAL GAS 5 =,F13.5) 01 FORMAT(13H S.S. FOR S3) 02 FORMAT(13H S.S. FOR S4) 13 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H ERMx=,E16.6,1x,5HERMN=,E16.6,1x,5HDLMX=,E16.6,1x,5HDLMN= 1,E16.6,1x,5HDLP =,E16.6,1x,5HTMIN=,E16.6) 000 FORMAT(11H S.S. FOR A) 4 FORMAT(6D13.5) 66 FORMAT(6D13.5) 66 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) 14 FORMAT(13H S.S. FOR BDC) 15 FORMAT(13H S.S. FOR BDC) 15 FORMAT(13H S.S. FOR BDC) 16 FORMAT(13H S.S. FOR BDC) 17 FORMAT(13H S.S. FOR BDC) 18 FORMAT(13H S.S. FOR BDC) 18 FORMAT(13H S.S. FOR BDC) 19 FORMAT(13H S.S. FOR BDC) 19 FORMAT(13H S.S. FOR CS) 19 FORMAT(13H S.S. FOR BDC) 10 FORMAT(13H S.S. FOR BDC) 10 FORMAT(13H S.S. FOR BDC) 10 FORMAT(13H S.S. FOR S1, S) 10 FORMAT(13H S.S. FOR BDC) 11 FORMAT(13H S.S. FOR BDC) 12 FORMAT(13H S.S. FOR BDC) 13 FORMAT(13H S.S. FOR BDC) 14 FORMAT(13H S.S. FOR BDC) 15 FORMAT(13H S.S. FOR BDC) 16 FORMAT(13H S.S. FOR BDC) 17 FORMAT(13H S.S. FOR BDC) 18 FORMAT(13H S.S. FOR BDC) 19 FORMAT(13H S.S. FOR S1, S) 10 FORMAT(13H S.S. FOR BDC) 10 FORM</pre>
<pre>1,13, 5x,7HMCOUNT=,[3, 5x,6HNPRNT=,[3,5x,6HICOND=,13,5x,4HIBC=,[3] 22 FORMAT(11HOFOR TIME =,E13.5,5x,13HTOTAL GAS 3 =,E13.5,4x,13HTOTAL 1GAS 4 =,F13.5,4x,13HTOTAL GAS 5 =,F13.5) 01 FORMAT(13H S.S. FOR S3) 02 FORMAT(13H S.S. FOR S4) 13 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H ERMx=,E16.6,1x,5HERMN=,E16.6,1x,5HDLMX=,E16.6,1x,5HDLMN= 1,E16.6,1x,5HDLP =,E16.6,1x,5HTMIN=,E16.6) 000 FORMAT(11H S.S. FOR A) 4 FORMAT(6D13.5) 66 FORMAT(6D13.5) 66 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) 14 FORMAT(13H S.S. FOR BDC) 15 FORMAT(13H S.S. FOR BDC) 15 FORMAT(13H S.S. FOR BDC) 16 FORMAT(13H S.S. FOR BDC) 17 FORMAT(13H S.S. FOR BDC) 18 FORMAT(13H S.S. FOR BDC) 18 FORMAT(13H S.S. FOR BDC) 19 FORMAT(13H S.S. FOR BDC) 19 FORMAT(13H S.S. FOR CS) 19 FORMAT(13H S.S. FOR BDC) 10 FORMAT(13H S.S. FOR BDC) 10 FORMAT(13H S.S. FOR BDC) 10 FORMAT(13H S.S. FOR S1, S) 10 FORMAT(13H S.S. FOR BDC) 11 FORMAT(13H S.S. FOR BDC) 12 FORMAT(13H S.S. FOR BDC) 13 FORMAT(13H S.S. FOR BDC) 14 FORMAT(13H S.S. FOR BDC) 15 FORMAT(13H S.S. FOR BDC) 16 FORMAT(13H S.S. FOR BDC) 17 FORMAT(13H S.S. FOR BDC) 18 FORMAT(13H S.S. FOR BDC) 19 FORMAT(13H S.S. FOR S1, S) 10 FORMAT(13H S.S. FOR BDC) 10 FORM</pre>
<pre>22 FORMAT(615) FORMAT(11HOFOR TIME =,E13.5,5X,13HTOTAL GAS 3 =,E13.5,4X,13HTOTAL 1GAS 4 =,F13.5,4X,13HTOTAL GAS 5 =,E13.5) OL FORMAT(13H S.S. FOR S3) D2 FORMAT(13H S.S. FOR S4) D3 FORMAT(13H S.S. FOR S5) 33 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H EKNX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTMIN=,E16.6) OOO FORMAT(11H S.S. FOR A) 44 FORMAT(6D13.5) 66 FORMAT(6D13.5) 66 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) ************************************</pre>
<pre>22 FORMAT(615) FORMAT(11HOFOR TIME =,E13.5,5X,13HTOTAL GAS 3 =,E13.5,4X,13HTOTAL 1GAS 4 =,F13.5,4X,13HTOTAL GAS 5 =,E13.5) OL FORMAT(13H S.S. FOR S3) D2 FORMAT(13H S.S. FOR S4) D3 FORMAT(13H S.S. FOR S5) 33 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H EKNX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTMIN=,E16.6) OOO FORMAT(11H S.S. FOR A) 44 FORMAT(6D13.5) 66 FORMAT(6D13.5) 66 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) ************************************</pre>
100 FURMAT(11HOFUR TIME =,E13.5,5X,13HTUTAL GAS 3 =,E13.5,4X,13HTUTAL 10A5 4 =,E13.5,4X,13HTUTAL GAS 5 =,E13.5] 101 FURMAT(13H S.S. FUR S3) 102 FURMAT(13H S.S. FUR S4) 103 FURMAT(13H S.S. FUR S5) 133 FURMAT(14H S.S. FUR S5) 133 FURMAT(16H ERNX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTM1N=,E16.6) 000 FURMAT(14H S.S. FUR A) 44 FURMAT(6D13.5) 155 FURMAT(16D13.5) 166 FURMAT(13H S.S. FUR CS) 77 FURMAT(13H S.S. FUR BDC) 10000 FURMAT(13H S.S. FUR BDC)
<pre>1GAS 4 =,F13.5,4X,L3HTDTAL GAS 5 =,F13.5) 01 FORMAT(13H S.S. FOR S3) 02 FORMAT(13H S.S. FOR S4) 03 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H ERMX=,F16.6,1X,5HERMN=,F16.6,1X,5HDLMX=,F16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,F16.6,1X,5HTM1N=,F16.6) 000 FORMAT(11H S.S. FOR A) 4 FORMAT(6E13.5) 66 FORMAT(6E13.5) 66 FORMAT(6E13.5) 66 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) 1000 FORMAT(13H S.S. FOR BDC) 1000 FORMAT(13H S.S. FOR BDC) 1100 FORMAT(13H S.S. FOR BDC) 111 DATA***********************************</pre>
<pre>D1 FORMAT(13H S.S. FOR S3) D2 FORMAT(13H S.S. FOR S4) D3 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H ERMX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTM1N=,E16.6) D000 FORMAT(11H S.S. FOR A) 44 FORMAT(6D13.5) 55 FORMAT(6D13.5) 66 FORMAT(6D13.5) 0000 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) ************************************</pre>
Ho2 FORMAT(13H S.S. FOR S4) HO3 FORMAT(13H S.S. FOR S5) HO3 FORMAT(6H ERMX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTMIN=,E16.6} HO3 FORMAT(6H ERMX=,E16.6,1X,5HTMIN=,E16.6) HO4 HORMAT(11H S.S. FOR A) 44 HORMAT(6D13.5) H06 FORMAT(6D13.5) H06 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) H000 FORMAT(13H S.S. FOR BDC) H07 FORMAT(13H S.S. FOR BDC) H08 FORMAT(13H S.S. FOR BDC) H09 FORMAT(13H S.S. FOR BDC) H00 FORMAT(15,52,52,52,52,52,52,52,52,52,52,52,52,52
103 FORMAT(13H S.S. FOR S5) 33 FORMAT(6H ERNX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTM1N=,E16.6) 000 FORMAT(1H S.S. FOR A) 44 FORMAT(6D13.5) 66 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) 0000 FORM
<pre>33 FORMAT(6H ERMX=,E16.6,1X,5HERMN=,E16.6,1X,5HDLMX=,E16.6,1X,5HDLMN= 1,E16.6,1X,5HDLP =,E16.6,1X,5HTM1N=,E16.6) 000 FORMAT(11H S.S. FOR A) 4 (DRMAT(6D13.5) 66 FORMAT(6D13.5) 66 FORMAT(6E13.5) 000 FORMAT(13H S.S. FOR CS) 7 FORMAT(13H S.S. FOR BDC) 000 FOR BDC) 000 FOR BDC 000 FOR B</pre>
1,E16.6,1X,5HDLP =,E16.6,1X,5HTMIN=,E16.6) 000 FURMAT(11H S.S. FOR A) 44 FORMAT(620.6) 55 FORMAT(6013.5) 66 FORMAT(613.5) 000 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) ************************************
000 FURMAT(11H S.S. FUR A) 44 FURMAT(4E20.6) 55 FURMAT(6D13.5) 66 FURMAT(6D13.5) 000 FURMAT(13H S.S. FUR CS) 77 FURMAT(15,5E15.6) 000 FURMAT(13H S.S. FUR BDC) ************************************
<pre>44 LORMAT(4E20.6) 55 FORMAT(6D13.5) 66 FORMAT(6D13.5) 600 FORMAT(13H S.S. FOR CS) 77 FORMAT(13H S.S. FOR BDC) 6000 FORMA</pre>
55 FURMAT(6D13.5) 66 FURMAT(6E13.5) 000 FURMAT(13H S.S. FUR CS) 77 FURMAT(13H S.S. FUR BDC) 000 FURMAT(13H S.S. FUR BDC)
66 FORMAT(6E13.5) 000 FORMAT(13H S.S. FOR CS) 77 FORMAT(15,5E15.6) 0000 FORMAT(13H S.S. FOR BDC) **********READ ALL (NPUT DATA**********************************
000 FORMAT(I3H S.S. FOR CS) 77 FORMAT(I5,5E15.6) 000 FORMAT(I3H S.S. FOR BDC) ************************************
77 FORMAT(15,5E15.6) 0000 FORMAT(13H S.S. FOR BDC) ************************************
<pre>iooo FURMAT(13H S.S. FUR BDC) ix************************************</pre>

READ (5,1000) N, NE, MOP, MER, MCUUNT, NPRNT, ICOND, IBC READ (5,2000) E, R, F1, F2, F3, CS1, CS2, TEND, DEL, BD, ERMX, ERMN, DLMX, DLMN, IDLP, TMIN
READ (5,2000)E,R,F1,F2,F3,CS1,CS2,TEND,DEL,B0,ERMX,ERMN,DLMX,DLMN, 1DLP,TM/N
IDLP, TM/N
READ(5,1001)(X(J),1=1,N),G

WRITE(6,2010)
WRITE(6,2070)N, NE, MUP, MER, MCDUNT, NPRNT, ICOND, IBC
WRITE(6,2020)E,R
WR1FE(6,2030)F1,F2,F3, CS1,CS2
WRITE(6,2040)TEND, DEL, BD
WRITE(6, 3333)ERMX,ERMN,DLMX,DLMN,DLP,TMIN
wRITE16,2050)(1,X(I),I=1,N)
WR11E(6,2055)G
IF(MCOUNT.EQ.01GO TO 5

WRITE(7,222)N, NE,MOP,MER,MCOUNT,NPRNT
WRITE(7,44441E,R.F1,F2,F3, CS1,CS2,TEND,DEL,ERMX,ERMN,DLMX,DLMN,
IDLP, TMIN
WRITE(7,5555)X,G
«★☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆☆
5 I=0.

L N = N - 1
NP 2=N+2
NSQ=N*N
NP I = N + 1
API=NPI
**********CALCULATE INITIAL POLYNOMIAL TEMPERATURE COEFFICIENTS*****
A(1)=0.
A(2) = 0.
DQ 20 J=1,LN
$90 \ 15 \ 1 \approx 1 \times N$
K = J + (I - I) + N
$K = J + (1 - 1) \times (1 + 1)$ B(K) = X(J + 1) × (1 + 1)
$L = J \times N$
B(I, I) = J + I
AB(J)=0.

1234567890123456789012345678901234567890123456789012345678901234567890123456789012345 CARD 0111 20 CONTINUE 0112 B(NSQ) = NP1AB(N)=1. 0113 0114 CALL SIMU(8,A8, N,KS) IF(KS.NE.0)WR11E(6.4000) 0115 00 25 I=1, N 0116 A(I+2)=AB(1) 0117 *********SET INITIAL LOCAL SOLID SPECIES VALUES********************* 0118 (×××× C1(1) = 10119 0120 C2(1)=0. 0121 C6([]=0. 0122 25 CONTINUE 0123 0124 С 0125 CALL STARTINE, MOP, MER, DEL, ERMX, ERMN, DLMX, DLMN, DLP) 0126 TPRT=0. 0127 NCUUNT=0 0128 THE VARIABLES AT THE NEXT TIME STEP ********************* 0129 0 0130 31 CALL KAMSUB(1) DELT=T-TPRT 0131 NCOUNT=NCOUNT+1 0132 C**************F THE TOTAL NUMBER OF TIME STEPS OR THE TOTAL LENGTH OF C TIME SINCE THE LAST PRINTED STEP IS THE DESIRED VALUE, C THEN THE CALCULATIONS WILL PROCEED, OTHERWISE THE PROGRAM 0133 0134 C. 0135 C WILL RETURN TO INTEGRATE FOR THE NEXT TIME STEP.********** 0136 Ċ 0137 IF (NCOUNT.GE.NPRNT.OR.DELT.GE.TMIN) GD TO 32 0138 GU TO-31 0139 32 TPRT=1 C***********USING THE NEW SOLID SPECIES VALUES, CALCULATE THE LOCAL C_______VALUES FOR EACH GAS SPECIES, MASS FLUX AND TOTAL SOLID AS C_______WELL AS THE POLYNOMIAL COEFFICIENTS FOR THE OVERALL 0140 0141 0142 0143 AVERAGE GAS SPECIES, MASS FLUX AND SOLID WEIGHT.********* C 0144 CS(1) = CI(1) + C2(1) + C6(1)0145 CS(2) = 0. NCOUNT=0 0146 xp=c6(1)/cs1/cs2+c2(1)/cs1 0147 $C3(1) = \{1, -CS1\} * XP$ 0148 C4(1) = 1, -C1(1) - XP0149 C5(1)=(1.-CS2)*C6(1)/CS2 0150 0151 RCS(1) = CI(1) + C2(1) + C6(1)0152 SSEX(1) = -OSM(1)0153 DD 140 J=2,N 0154 SSFX(J)=-DSM(J) 0155 XPTH=C6(J)/CS1/CS2+C2(J)/CS C3(J)=(1.-CS1)*XPTH 0150 0157 C4(J) = 1 - C1(J) - XPTH0158 C5(J)=(1.-CS2)*C6(J)/CS2 0159 (L)X=LX ASUM=0. 0160 DU 135 I=1,NP2 0161 1M1 = 1 - 10162 *******CALCULATE THE LOCAL TEMPERATURE USING THE NEW TEMPERATURE 0163 C * 0164 έC. ASUM=ASUM+A(I)*X(J)**1M1 0165

80/80 LIST

		00001111111112222222222333333333334444444444	
	123450	<u>66789012345678901234567890123456789012345678901234567890123456789012345</u>	6789012345
CARD			
0166		IF(1.GT.LN)G0 TO 135	
0167		IP1=I+1	
0168		K=(J-1)+IMI*LN	
0169		X11=x1v=11X	
0170		BCS(K)=XJ1	
0171		BDC(K)=XJI	
0172		BS3(K)=XJI	
0173		B\$4(K)=XJI	
0174		BS5(K)=XJI	
0175	135	CONTINUE	
0176		1 一 し 一 1	
0177		RCS(J) = C1(J) + C2(J) + C6(J)	
0178		BBS3(JM1) = C3(J) - C3(1)	
0179		BBS4(JN1)=C4(J)-C4(1)	
0180		BBS5(JM1)=C5(J)-C5(1)	
0181		BBCS(JM1)=RCS(J)-RCS(1)	
0192		BBDC(JM1) = DSM(J) - DSM(1)	:
0183		TMPTR (J) = ASUM	
0184	140	CONTINUE	
0185		FSUM=0.	
0186		CSUM=0.	
0187		IF(LN.EQ.1)GO 10 142	
0188		CALL SIMQ(BS3,BBS3,LN,KS)	
0189		1F(KS.NE.O)WRITE(6,3001)	
0190		CALL SIMQ(BS4, BBS4, LN, KS)	
0190		1F(KS.NE.O)WR1TE(6,3002)	
0192		CALL SIMU(BS5,BBS5+LN+KS)	
0192	· · · · · · · · · · · · · · · · · · ·	1F(KS.NE.0)WRITE(6,3003)	
0194	1 - A	CALL SIMG(BCS, BBCS, LN, KS)	
0195		[F(KS.NE.0)WRITE(6,7000)	
. 0195		CALL SIMO(BDC, BBDC, LN, KS)	
0198		IF(KS.NE.0) WRITE(6,8000)	
	14.3	Prixs.Ne.01WK11E10,00005	
0198	147	173 143 143 121, LN 1P2=1+2	······································
		AP 2 = 1 + 2	
0200		APZ-1FZ CS(1P2)=EBCS(1)	
0202 -		C S (1P27-C) (S (1P2)/AP2	
0202	· · · · · · · · · · · · · · · · · · ·	DCT(1P2)=BBDC(1)	
		S3(1P2)=8BS3(1)	
0204		S4(1P2)=BBS4(1)	
0205		S5(1P2)=BBS5(1)	1. A.
0206		S5(1P21=HD55(1) FSUM=FSUM+DC1(1P2)/AP2	· · · · · · · · · · · · · · · · · · ·
0207	1/2	FSUM=FSUM+DCT(IP2J/AP2 B CUNTINUE	
0208	143		
0209		DCT(1)=DSM(1)	
0210		DCT(2)=0.	
0211		FLUX=-FSUM-DCT(1)	
0212		WGT=CSUM+CS(1)	
0213		S3())=C3())	
0214		<u>S4(1)=C4(1)</u>	·
0215		S5(1)=C5(1)	
0216		53(2)=0,	
0217		\$4(2)=0.	
0218		\$5(?)=0.	
0219		TMPTR(1) = A(1)	
0220		SPC3=0.	

80/80_LIST

	000000000111111111222222222233333333334444444444
	123455789012345678901234501234567890123458
CARD	
0221	SPC+=Q.
0222	SP(0.5 = 0)
0223	$f(t = 1.4.1 \pm 1.5.0)$
0224	() J = L
0225	
0276	SPC6=SPC6+ S6(T)/D1
0227	SPC5=SPC5+ 55(1)/01
0228	141 CUNTINUE
0229	IFIMCHURI.CO.0100 TO 33
0230	- (φασφφασφφαφαφαφέει- DESIRED, PUNCH ()UTPUTφφφαφφφφφφφφφφφφφφφφφφφφφφφφφφφφφφφ
0231	NR11E(7,0660)T,SPC3,SPC4,SPC5,FLUX,WGF
0232	WRITE(7,777)(J,FMPTK(J),RCS(J),CI(J),C2(J),C6(J),C3(J),C4(J),C5(J
0233	
0234	(作业公共会会会办办会会办的历代(101、1011月11日,各社会资源的学校会会合作的公共会会合合会会会会会会会会会会会会会会会会会会会会会会会会会会会会会会会会会
0235	33 WRITEL6.3000) T.SPC3,SPC4,SPC5
0236	- WKIIF(6,1010)
0231	WRITE(6,111)(J,TMPTR(J),RCS(J),C1(J),C2(J),C6(J),C3(J),C4(J),C5(J)
0248	1), SS1-X(J), J=1,N)
0234	WRITE(6,1500))LUX,WGF
0240	C*************************************
0241	() [] [] [] [] · · · · · · · · · · · · · ·
0242	IF (T.GE. TEND) STUP
0243	G0 10 31
0244	END

80/80 LIST

	00000000111111111222222222333333333334444444444	
	<u>12365678901236567890123456789012345678901234567890123456789012345678901234567890</u>	12345
CARD		
0001	SUBKOUTING DERFUN	
0002	· (\$\$\$77 \$\$4 \$\$\$\$\$\$ \$\$4 \$\$\$\$\$\$\$\$\$\$\$\$\$\$\$\$	* * *
E 000	<u>C # +</u>	**
0004	C**THIS ROUTINE IS THE COMPANION PROGRAM FOR THE INTEGRAL METHOD	* *
0005		# #
0006	C**	**
0007	<u>C # +THE STAULTANEOUS ORDINARY DIFFERENTIAL EQUATIONS ARE FORMED AND</u>	* *
0008	C**SOLVED TO GIVE A SYSTEM OF EXPLICIT FIRST ORDER ORDINARY	**
0009		**
0010		* *
	COMPUT IS THE VARIABLES AT THE NEW TIME STEP AND COMES FROM THE	**
0012	C##INTEGRATION ROUTING.	**
		**
0013		¥¥
-0014	C**OUTPUT 15 THE DERIVATIVES OF THE VARIABLES AT THE SAME TIME STEP	
0015	C**AND REFURNS THEM TO THE INTEGRATION ROUTINE	**
0016	C * *	* *
	_C**DIMENSIONED_AND_EQUIVALENCED_VARIABLES_CHANGE_WITH_THE_ORDER_OF_TH	
0018	C**APPROXIMATION AND CAN BE INSERTED AS A GROUP BETWEEN THE FIRST	ጥጥ
	<u>C**DUBLE PRECISION STATEMENT AND THE SECOND CONTINUATION CARD OF THE</u>	
0020	C**EQUIVALLNCE STATEMENT.	* *
0021	<u>C++</u>	**
0025	C##THE ROUTINE ISING! SULVES A SET OF SIMULTANEOUS ALGEBRAIC FQUATION	5 * *
.0023	CERAND WILL PRINT AN S.S. MESSAGE IF THE MATRIX IS SINGULAR.	¢Φ
0024	C * *	**
.0025	· C 各种体质和分析化、自己的有效体质的合体,在全体的体质、全体的体质和分析和分析和分析和分析和分析和分析和分析和分析和分析和分析和分析和分析。	**
0026	COMMON Y (202)	
0027	DOUBLE PRECISION DEXP.DLQG	
0029	DOUBLE PRECISION G, BMN, ASUM, BSUM, CSUM, TSLP, DRG, HS(3, 7), CNDX,	
0029	18N, BG(25), BH1(49), BH2(25), BH3(25), BBG(5), BBH1(7), BBH2(5),	
0030.	268H315), BM1, GS(6), RCS(6), SUBK(3,6), Q(3), X(6), CK1(6), CK2(6), CK3(6)	
0031	16F(6), BT(25), BBT(5)	
0032	COMMEN/COMA/X, G, E(3), R(3), FLUX, LN, NSQ, N, NE, FI, F2, F3, CS1, CS2	
0033	LINPZ WPL WGT ICOND, IBC, BD, DSM(6), APL	
0034		
0035	EQUIVALENCE (Y(1), C1(1)), (Y(7), C2(1)), (Y(13), C6(1)), (Y(19), A(1))	ц х .
0036	1(Y(2/), T), (Y(29), DC1(1)), (Y(35), DC2(4)), (Y(41), DC6(1)), (Y(47), D	
0037		
0038	5500 FORMAT(12H S.S. EOR GS)	
0039	6000 FORMAL(12H-S.S. FOR GT)	
0040	6500 FURMAT(13H S.S. FOR HS1)	
0041	6501 FORMAT(13H S.S. FOR HS2)	
0042	6502 FURMAT(13H S.S. FOR HS3)	
0043	9500 FORMAT(12H S.S. FOR DA)	
0044	C##############CALCULATE THE RATIO OF THE HEAT FLUX PARAMETER TO THE	
0045	<u>C</u> <u>PRE-EXPUNENTIAL</u> PARAMETER***********************************	(X: 10
0046	Q(t) %R(t)/F1	
0047	<u>0(2)</u> ⇒ <u>R(2)/F2</u>	
0048	U(3):R(3)/F3	
0049	AN≠N	
0050	GS(1) = 0.	
0051	C#############CALCULATE THE REACTION RATE CONSTANTS AND THE POLYNOMIAL	-
0052	C CUEFFICIENTS FOR HEAT GENERATION INTEGRAL***************	
0053		
0054	00.40.1=1.2	
0055	SUBK(I,1)=DFXP(-E(1)/(1.+G*A(1)))	

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1	
	00000000111111111222222233333333334444444444
6400	<u>1234567890123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890123456</u>
CARD	
0056	$HS(1,1)=C1(1) \neq SUBK(1,1)/RCS(1)$
0057	HS(1,2)=0.
0058	40 CUNTINUE SUBK(3,1)=DEXP(-E(3)/(1.+G*A(1)))
0059	HS(3,1)=C2(1)*SUBK(3,1)/RCS(1)
0061	HS(3,21=0.
0062	CK1(1)=C1(1)*SUBK(1,1)/F1
0063	CK(2(1) = C1(1) + SUGK(2,1) / F2
0064	$C(3)(1) = C^2(1) * SUBK(2)(1)/F 3$
0065	DC1(1) = -CK1(1) - CK2(1)
0066	$DC2(1)=CS1^{+}CK1(1)+CK3(1)$
0067	DC6(1) = C52 * CK3(1)
0068	DSM(1) = DC1(1) + DC2(1) + DC6(1)
0069	GS(1) = DLOG(RCS(1)) * 2 * A(3)
0070	GF(1) = 0.
0071	ANP2 = A(NP2)
0072	$DO_{-60,-j=2,N}$
0073	ASUM=0.
0074	BSUM=0.
0075	CSUM=0.
0076	xJ=x(J)
0077	00 45 1=1,NP2
0078	IMI=I-1
0079	1M1**tCX
0080	CSUM=CSUM+A(I)*XJ11
0081	IF(1.6T.LN)GU TO 45
0082	IP 2=1+2
0083	B1=T+1
0084	$\beta [1 \neq 1 \neq (1+1)]$
0085]**LX=LVX
0086	x JP1=x J**(1+1)
0087	A SUM=A SUM+B I*A (IP2)*XJI
00.88	BSUM=BSUM=BIL*A(IP2)*XJFI
0089	K = (J - L) + I M L + L N
0090	
0091	
0092	BG(K)=XJI BT(K)=XJI
0093	
0094	BH1(K) = XJP1 BH2(K) = XJP1
0096	BH3(K) = XJP1
0097	45 CONTINUE
0098	A SUM+ A SUM+ A P L * A NP 2 * X J * * N
0099	BSUM=BSUM+AN*AP1*ANP2*XJ**(N~1)
0100	
0101	RCS(J) = C1(J) + C2(J) + C6(J)
0102	DR = DL OG (RCS [J])
0103	BBG(JMI)=DRG+BSUM-GS(I)
0104	BBT(JMI)=DRG*ASUM
0105	p_{0} 50 L=1,3
0106	SUBK(L,J) = DEXP(-E(L)/(1.+G*CSUM))
0107	50 CONTINUE
0108	BBH1(JM1)=C1(J)*SUBK(1,J)/RCS(J)-HS(1,1)
0109	BBH2(JM1) = C1(J) + SUBK(2,J) / RCS(J) - HS(2,L)
0110	BBH3(JM1)=C2(J)*SUBK(3,J)/RCS(J)-HS(3,1)
والمتحدث والمتحد والم	

80/80 LIST

	00001111111112227222223333333333344444444455555555555
	<i>b</i> 7 <u>8</u> 401 <u>234567890123456789012345678901234567890123456789012345678901234567890123456</u>
CARD	
	, CK1(J)=C1(J)*SUBK(1,J)/F1
0112	CK2(J)=C1(J)*SUBK(2,J)/F2
0113	CK3(J) = C2(J) + SUEK(3, J)/F3

	DC1(J) = CK1(J) = CK2(J)
)116 117	DC2(J)=CS1*CK1(J)~CK3(J)
/#4(DC6(J)=CS2*CK3(J) DSM(J)=DC1(J)+DC2(J)+DC6(J)
118 0119 60	CONTINUE
0120	If (LN.E0.1)60 TO 99
121	CALL SIMO(06,800,10,KS)
122	If (KS.NE.0) wRITE (6,5500)
	CALL SIMU(BT, BBT, LN, KS)
124	IF(KS.NE.0)WRITE(6,6000)
125	CALL SIMO(BH1, BBH1, LN , KS)
126	IF(KS.NE.0)WRITE(6,6500)
127	CALL SIMQ(BH2, BBH2, LN ,KS)
123	IF (KS.NF.0) WRITE $(a, a501)$
129	CALL SIMQ (BH3, BBH3, LN ,KS)
130	IF(KS.NE.0)WRJTE(6,6502)
131 90	CONTINUE
132	ASR+=0.
133	QQ 95J⊎lilaN ≤
134	_1P1=1+1
	GS(1P1)=88G(1)
	<u>GT(1P1)=88T(1)</u>
138	HS(1,1P2)=88H1(1)
132	MS(2+1P2)=000(2(1))
140	N5(3, 192)=86H3(1)
141	ASRE=ASRE+A(1P2)
	CONTUNE A SEE A SUG LA A NA A AND A
	_ASRF=ASRF+A(1)+A(NP2) #*#######CALCULATE_POLYNOMIAL_COEFFICIENTS_FOR_THE_TEMPERATURE
145 6	DERIVATIVE WITH CHOICE OF BOUNDARY CONDITIONS AND THERMAL
146 0	CONDUCTIVITY EFFECTS**********************
147	DNDX=DSM(N)
148	CNDX=RCS[N]
149	[F(1CUND.EQ,14GH T0 96
150	RC5(N)=1.
151	9749 X = 0
152	IF (ICUND.E. u . OICNDX=1.
153 96	182=QS(T)
154	IF(IBC.EQ.F)TP2=(1BU#ASRF)
155	TSLP=(1.+DLOG(CNDX)) # P2/RCS(N)
156	BRCS=BOZRCS(N)
157	0() 110 M≈1, N
158	ASUM=0.
	.8SUM≖0.
The second se	©SUM≠9.
160	
160	0.5UM=0.
159 160 16]	841=4
160	

80780 EIST

80780 E1ST

		000011111111122255						
	12345	6739012345673901234	15678901234	156789012	234567890	1234567890	12345678	90123458
CARD								
0166		CSUM=CSUM+QLJ1#HS	(J, [)/BN					
0167	100	CONTINUE						
01.5 P		11(1.GT.N)G0 TO 10	<u>}</u>					
01697		BMN≈M+T+1		• •				
0170	1.1	HM3=1+1						
0171	1997 - 19	BSUM=BSUM+BM3*ALE4	•2)ZBN					
0172		K=M+I #NP1						
0173		BHILK)=1./HMN						
0174		IF(ICOND.E3.0)60						
0175		ASUM=ASUM+GS(1)/BF						
0176	1.11	1+(1.GT.LN)60 10 1						
01//		DSUM=DSUM+G1(1+1)/	' BN					
0178	105	CUNTINHE						
0179		BH1(N)=1./AM1						
0180		$\mathbf{I} = (\mathbf{M} + \mathbf{I}) + \mathbf{N} \mathbf{P} \mathbf{I}$				· · · · · · · · · · · · · · · · · · ·		
0181		XX1=H+1				· · · · · ·		
0132	1. A.	[[[[]XX]=X)	(I+BRCS					
0163		BH1(L)=XX1						
0184	5 L L	BM2=M-1						
0135		BBHL(M)=TSLP-BM2*L	1204-7204-6	IM2*05UM-	-0.50476		1	
01.86	110	CONTINUE	ant mpan at lar na an 1971 a na 1971 a 1971 a 1971 a 1971 a	·				
01-37	i ta s	TP1=0.						
0138		TP3=0QS(1)						
0139		IF(IBC.EQ.0)60 F0	111					
0190		TP3=0.						
0191		1P1=BRCS						
0192 -	111	BH(I(NP1)=TP1		erra a a a a		·····		
0193 -		BBHE(NP1) = -TP2 * DHf		TRUSTAL	FIPSZRUSI	- 10		
01.94		CALL SIMULBHL, BBHD						
0195		IF(KS.NE.O)WRITE(0,45001					
0196		DO1 115 1=2,NP1						
0197	÷	DA(1+1)=BBH1(1)						
0198	. 115	CUNTINUE		د ام مادی منه – از المنابط دارد می رس				
0128	·	DA(1) = BoH1(1)						
0200		DA(2)=0.	· ·					
10201		RETURN						
0505		END					· · · · ·	
0203		FUNCTION OSITI						
0204		<u>05=1.</u>						
0205		RETURN						
0509		END						
0207		FUNCTION DOS(T)						
0509		DQS=0.						
0508		RETURN						
0210		END						

.

1234567891123456789012345678901234567890123456789012345678901234567890123456 CARD SUBROUTINE SIMULA, B, N, KSI 0001 SIM DOUBLE PRECISION DABS 00.92 0003 DOUBLE PRECISION BIGA, SAVE, A(1), B(1) SIM 0004 C SIML 0005 0 - FORWARD SOLUTION SIM. 0006 C SIM 0007 101-0.0 SIM 0008 85=0 SIM. 0009 SIMU JJ=-14 00 65 J=1,N SIMU 0010 JY=J+I SIM. 0011 SIMO 0012 1J=11+N+1 0013 BIGA≈O SIMC 0014 1[=1]-3 SINC 0015 DO 50 1=1.N SIM 0016 -C SIM SEARCH FOR MAXIMUM COEFFICIENT IN COLUMN SIM 0017 C C \$13 0018 SIM 11=11+1 0019 IF((DABS(BIGA)-DABS(A(IJ))).GE.0.1G0 TO 30 SIM 002.0 BIGA=A(1J)SIM 0021 SIMI 0022 IMAX = I 30 CONTINUE SIMI 0023 0024 C. SIMU IFST FOR PIVOT LESS THAN TOLERANCE (SINGULAR MATRIX) 0025 С SIMO 0026 SIM Ċ 0027 IF ((DABS(BIGA)-IDL).GT.0.)GD TO 40 SIM SIM. 0028 KS=1 RETURN SIM 0029 SIM 0030 ĩ INTERCHANGE ROWS DE NECESSARY 0031 SIMU С SIM 0032 C 40 11=J+N*(J-2) STM 0033 SIM 0034 11=1MAX-J 00 50 K=J;N SIME 0035 S.LM. 0036 11=11+N SIM: 0037 12=11+11 0038 SAVE=A(11) SIM 0039 A(11)=A(12) SIM 0040 A1121=SAVE 51 M 0041 C DIVIDE EQUATION BY LEADING COEFFICIENT SIM. Ċ 0042 SIM Ċ 0043 SIM 50 A(11)=A(11)/dIGA 0044 SIM 0045 SAVE=B(IMAX) SIM 0041 H([MAX]=H[J])SEM 0047 BEJ)=SAVE/BIGA SIMI 0048 ΞĊ FLIMENATE NEXT VARIABLE 0.049 SIMI ¢ 0050 Ċ SIMI 0051 IF((J-N).FU.0)60 TO 70 SIM 0052 105=N+(J-1) SIM 00 65 1X=JY,N SIM 0053 1×3=105+1× SIM 0054 SIM 0055 1 T = J - I X

80780 LIST

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CARD	
	Ş
1 X J X = X X X = X X X = X X X X = X X X X	S
<u>0055</u> JJX=IXJX+II	<u>S</u>
00 A(IXJX)=A(IXJX)=(XLX)A=(XLX) = 0.000	S
0000	S
0061 C	S
	S
0063 C	S
0064 70 NY=N-1	<u>S</u>
$0065 \qquad 11 = 0 \neq N$	S
0066	S
0067 [A=II-J	S
0000 <u>10 - N - J</u>	S
0065 IC=N	S
0070 DU 30 K=1.J	<u> </u>
0071 $3(18)=B(18)-A(1A)*B(1C)$	S
.0072 [A=[A=N	S
0073 30 IC=IC-1	S
0074 RETURN.	S S
0075 END	5

80/80 LIST

	BO/BO LIST
	000000001111111112222222223333333334444444444
	<u>12345678901234567890123456789012345673901234567890123456789012345678901234</u>
CARD	
0001	SUBROUTINE KAMSUBINSTART
0002	COMMON Y(202)
0003	COMMON /SHARE/NN.SPACE, MODE, KKA, EIMAX, EIMIN, E2MAX, E2MIN, FACT
0004	DIMENSION DELY(4,100), BET(4), XV(5), FV(4,100)
0005	COMMON/INTDAT/2(5,202), IERR
0007	DOUBLE PRECISION YU(5,100) C
0008	
0009	
0010	JE (NSTART.LE.O)GO TO 9977
0011	IF (MODE.EQ.1) CO TO 1001
0012	GU TU 2000
0013	C RUNGE-KUTTA
0014	1000 LL = 1
0015	1001 x TM2= Y (N2)
0016	XTM3=XV(MM)
0017	DO 1034 K=1.4
0018	XTM1=BF1(K)
0019	00 1350 I=1,NN
0020	DELY(K,I)=XIM2 *FV(MM,I)
0021	$\frac{(\mathbf{Q} = \mathbf{Y} \mathbf{U} (\mathbf{M} \mathbf{M}, \mathbf{I}))}{(\mathbf{Q} = \mathbf{V} \mathbf{U} \mathbf{I} \mathbf{M} \mathbf{M} \mathbf{I})}$
0022	1350 Y(1)=Q+XTM1 *DELY(K, ()
0023	Y(NPL)=XTML*XTM2+XTM3 CALL DERFUN
0.024	$\frac{1100}{1100} I = 1.00$
0026	1PN2=1+N2
0027	1100 FV(M, 1)=Y((PN2)
0028	1034 CONTINUE
0029	MMP1=MM+1
0030	DO 1039 I=1,NN
0031	DEL=(DELY(1,1)+2.0*DELY(2,1)+2.0*DELY(3,1)+DELY(4,1))/6.0
0032	YU(MMP1,T)=YU(MM,L)+DEL
0033	Y(1)=YU(MMP1,1)
0.034	LO39 CONTINUE
0035	4M=AHP1
0035	XV(MN)=XV(MM-1)+Y(N2)
0037	Y(NPT) = XV(MM)
0038	CALL DERFUN
0039	IF(MODE.EQ.1)60 TO 42 DO 150 1≈1,NN
0.040	LPN2 = 1 + N2
0042	150 FV(MM, I) = Y(IPN2)
0043	IF(MM.NE.4)G0 TO 1001
0044	C ADAMS-MOULTON
0045	2000 XTM2=Y(N2)
0046	DO 2048 1=1,NN
0047	DEL=XTM2 *(55.0*FV(4,1)-59.0*FV(3.1)
0048	1+37.0*FV(2,1)-9.0*FV(1,1))/24.0
0049	Y(I)=YU(4,I)+DEL
0050	2049 DELY(1,1) = Y(1)
0051	Y(NP1) = XV(4) + XTM2
0052	CALL DERFUN
0053	XV(5)=Y(NP1) 00 2051 1=1,NN
0054	100 2051 t=1,NN IPN2=I+N2
0000	

	·	
		00000001111111112222222223333333334444444444
	CARD	1234567890123456789012345678901234567890123456789012345678901234567890123456789012345678901234
	0056	DEL=XTM2 *(9.0*Y(1PN2)+19.0*FV(4,1)
÷	0057	1~5.0*FV(3,[)+FV(2,])/24.0
	0058	YU(5, [)=YU(4, [)+DEL
	0059	2051 Y(I)=YU(5,I)
	0060	CALL DERFUN
	0061	IF (MUDF.NE.3) GO TO 42
·	0062	
	0063	\$SE=0.0
	0.064	DO 3033 1=1,NN
	0065	$EPSIL = R \times ABS(Y(1) - DELY(1,1))$
	0066	IF(KKA.FQ.2)GD TO 3307
	0067	IF(Y(1).EQ.0.)GD TO 3307
	0068	EPSIL=EPSIL/ABS(Y(I))
	0069	330/ IF((SSE-EPSIL).LT.O.)SSE=EPSIL
	0070	
	0071	IF((E1MAX-SSE).GF.0.)G0 T0 3035
: 	0072	IF((ABS(Y(N2))-E2MIN).GT.0.)GO TO 4340
	0073	GO TO 42
	0074	3035 IF((SSE-E1MIN).GE.0.)GO TO 42
	0075	IF((E2MAX-ABS(Y(N2))).GT.0.)GD TO 5360
	0076	<u>60 TO 42</u>
	0077	4340 LL=1
	0078	$1 \in RR = 1$
;	0079	MM=1
	0080	Y(N2)=Y(N2)*FACT
	0081	GU TO 1001
	0082	5360 IF(LL.EQ.1)GD 10 42
	600	XV(2) = XV(3)
	0084	XV(3) = XV(5)
	0085	DD 5363 1=1,NN
	0086	FV(2,1)=FV(3,1) TPN2=I+N2
	0087	FV(3, 1) = Y(1PN2)
	0089	YU(2,1)=YU(3,1)
	0090	5363 YU(3,I)=YU(5,I)
	0091	Y(N2)=2.0*Y(N2)
	0092	ERR = 2
	0093	
1	0094	im =3
	0095	CÚ 10 1001
	0096	C EXIT RUUTINE
	0097	42 IF(MODE.EQ.1)60 TO 43
	0098	DD 707 K=1,4
1	0099	Z(K, NP1) = XV(K)
1	0100	Z (K, N2) = XV (K+1) - XV (K)
	0101	DU 707 I=1,NN
	0102	Z(K,I)= YU(K,1)
2	0103	$1^{\text{PN2}} = \text{N2} + 1$
1	0104	707 Z(K, IPN2)= FV(K, I)
i.	0105	43 Z(5, NPI) = XV(5)
	0106	DO 708 I= L,NN
	0107	$2(5,1) \approx YU(5,1)$ 10N2 = N2 + 1
	0108	
	01:09	$709 \ 2(5, 1PN2) = Y(1PN2)$
	_D110	Z(5, N2) = Y(N2)

	0000	0000111111111122222222223333333334444444444
	12345	<u></u>
CARD		
0111		DD 12 K=1.3
0112		$X \vee (K) = X \vee (K+1)$
0113		00 12 1=1.NN
0114		FV(K,I)=FV(K+1,I)
0115	12	YU(K,[]=YU(K+],[]
0116		LL =2
0117		MM = 4
0118		XV(4)=XV(5)
0119		DD 52 J=1.NN
0120		IPN2=I+N2
0120		$1 \ge 1 \le $
0122	52	YU(4,1)=YU(5,1)
	52	
0123		IF(MODE, EQ.3)GO TU 73
0124	0010	RETURN
0125	9977	
0126		XV(2)=0.
0151	• • • • • • • • • • • • • • • • • • • •	×∨(3)=0.
0128		IERR = 3
0129		AL PHA = Y (NN +])
0130		- EPM=0.0
0131		MM±1
0132		1F(M00€.EQ.1)MM=4
0133		BF1(1)=0.5
0134		BET(2)=0.5
0135		B(T(3)=1,0)
0136		BET(4)=0.0
0137		N 2 = N N + 2
0138		Y(N2)=SPACE
0139		NP1=NN+1
0140		R=19.0/270.0
0141		XV(NM)=Y(NP1)
0142		<pre>If (F1MIN.LF.O.) & LMIN≈E1MAX/55.0</pre>
0143		IEIFACI, LE. O. IFACT=0.5
0144		CALL DEPFUN
0145		DU 320 I=1,NN
0146		IPN(2 = 1 + N2)
0147		FV(2,1)=0.
	· · · · · · · · · ·	FV(3,1)=0.
0148		
0149		$\frac{FV(MM_1)=Y(1PN_2)}{FV(MM_1)}$
0150		YU(2,1)=0.
0151		YU(3,1)≈0.
0152	320	YU(MM,T)=Y(1)
0153		<u>CO 10 1000</u>
0154	73	E=ABS(XV(4)-ALPHA)
0155		IF((E-EPM).LE.O.)GO TO 2000
0156		$[P M \neq E]$
0157		PETURN
0158		LND

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0001	SUBRUUFINE START1M1, M2, M3, A1, A2, A3, A4, A5, A6)
0002	CUMMUN/SHARE/NN,SPACE,MODE,KKA,EIMAX,EIMIN,E2MAX,E2MIN,FACT
0003	<u></u>
0004	C NO INFORMATION IS REQUIRED IN THIS SUBROUTINE
0005	NN≅M1 MODE≃M2
00007	KKA=M3
0008	Ε1ΜΔΧ = Α2
0009	E1MIN=A3
0010	£ 2MAX = A4
0011	<u>E2MIN=A5</u> FACT=A6
0013	SPACE=AI
0014	CALL KAMSUB(0)
0015	RETURN
0016	END
-	
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APPENDIX J

COMPUTER PROGRAM STEPS

Card Numbers	•	Step Description
MAIN		
73-76	(1)	Enter MAIN program and read the number of approxi- mation strips, N, and the program control constants.
78-85	(2)	Print all the input data.
86	(3)	Decide whether to punch the input data.
88-91	(4)	If desired, punch the input data.
93	(5)	Initialize time to zero.
95-99	(6)	Set internal program constants.
101-117	(7)	Calculate the initial polynomial temperature coeffi- cients (Equation 3.9).
119-122	(8)	Set the initial solid species densities (Equation 3.10).
125	(9)	Enter the ordinary differential equations integra- tion START program with its program control constants. Return to MAIN.
126-127	(10)	Initialize program constants.
130	(11)	Enter the ordinary differential equations integra- tion program KAMSUB to calculate the unknowns at the next time step. Enter subroutine DERFUN from KAMSUB to calculate the ordinary differential equations.
DERFUN		
46-50	(12)	Calculate program constants.
53-142	(13)	Calculate the reaction rate constants, the polyno- mial coefficients for the heat generation integral (Equation 3.5d), the polynomial coefficients for the

Card Numbers		Step Description
		integrals representing a change in the effective thermal conductivity (Equation 3.5b, c), and the local solid species derivatives.
143-200	(14)	Calculate the polynomial coefficients for the tem- perature derivative (Equation 3.6) including the surface boundary condition (Equation 3.7a or b) with terms depending on the type of boundary condition (Radiative or Convective) and case (0, 1, or 2) be- ing considered.
203-210	(15)	Give functions for time-dependent radiative heat flux and its derivative (constant flux considered).
201	(16)	Return derivatives of the variables to KAMSUB to be integrated for values at the new time step. Return values to MAIN.
MAIN		
137	(17)	If the total number of time steps or the total length of time since the last printed step is not the desired value, go back to KAMSUB and integrate for the next time step; otherwise continue with the values of variables at this time step.
144-228	(18)	Calculate the local values for each gas species, tem- perature, mass loss rate, and total solid; including the polynomial coefficients for the overall average values of each gas species, mass flux, and total weight.
229	(19)	Decide whether to punch the output.
231-233	(20)	If desired, punch the output.
235-239	(21)	Print the output.
242–243	(22)	If the desired total length of time has not been reached, go back to KAMSUB and integrate for the next time step; otherwise stop.
Note: Card	numbe	rs refer to the program listings given in Appendix I.

VITA 3

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