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FREE VIBRATIONS OF ORTHOTROPIC CONICAL  
SANDWICH SHELLS WITH VARIOUS BOUNDARY  
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WITH VARIOUS BOUNDARY CONDITIONS

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1969

FREE VIBRATIONS OF ORTHOTROPIC CONICAL SANDWICH SHELLS  
WITH VARIOUS BOUNDARY CONDITIONS

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## TABLE OF CONTENTS

	Page
ACKNOWLEDGEMENT . . . . .	iii
LIST OF TABLES . . . . .	iv
LIST OF FIGURES . . . . .	v
LIST OF SYMBOLS . . . . .	vi
 Chapter	
I. INTRODUCTION . . . . .	1
1.1 Survey of Shell Dynamics . . . . .	1
1.2 Research Objectives . . . . .	3
II. FORMULATION OF THE THEORY . . . . .	5
2.1 Method of Analysis . . . . .	5
2.2 Hypotheses . . . . .	5
2.3 Application of Galerkin's Method . . . . .	6
2.4 Boundary Conditions . . . . .	20
III. EVALUATION OF THE THEORY . . . . .	29
3.1 Homogeneous Cylinders . . . . .	29
3.2 Homogeneous Cones . . . . .	31
3.3 Sandwich Cone . . . . .	32
IV. CLOSURE . . . . .	44
REFERENCES . . . . .	46
 Appendix	
A. DERIVATION OF THE KINETIC AND POTENTIAL ENERGIES FOR AN ORTHOTROPIC SANDWICH SHELL . . . . .	50
A.1 Strain-Displacement Formulation . . . . .	50
A.2 Core Strain Energy . . . . .	54
A.3 Facing Strain Energy . . . . .	55

	Page
A.4 Total Strain Energy . . . . .	57
A.5 Total Kinetic Energy . . . . .	59
B. APPLICATION OF HAMILTON'S PRINCIPLE TO DERIVE THE EQUATIONS OF MOTION . . . . .	60
C. IDENTIFICATION OF INTEGRAL FORMS. . . . .	78
D. EVALUATION OF INTEGRALS FOR FREELY SUPPORTED BOUNDARY CONDITION . . . . .	85
E. EVALUATION OF INTEGRALS FOR CLAMPED-CLAMPED BOUNDARY CONDITION . . . . .	94
F. EVALUATION OF INTEGRALS FOR FREE-FREE BOUNDARY CONDITION . . . . .	97
G. COMPUTER PROGRAM DOCUMENTATION. . . . .	104
H. COMPUTER PROGRAM LISTING. . . . .	114

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## LIST OF TABLES

Table		Page
3.1	Frequencies for a Freely Supported Homogeneous Cylinder	30
3.2	Analytical Frequencies for Free-Free Homogeneous Cone	34
3.3	Analytical Frequencies for Freely Supported Sandwich Cone	36
3.4	Analytical Frequencies for Clamped-Clamped Sandwich Cone	38
3.5	Analytical Frequencies for Free-Free Sandwich Cone	41

## LIST OF FIGURES

Figure	Page
2.1 General Form of Stiffness Matrix . . . . .	19
2.2 Form of Stiffness Matrix when $n = 0$ . . . . .	21
2.3 Form of Stiffness Matrix for a Homogeneous Shell . . . . .	22
3.1 Natural Frequencies for a Homogeneous Cone . . . . .	33
3.2 Natural Frequencies for a Freely Supported Sandwich Cone	35
3.3 Natural Frequencies for a Clamped-Clamped Sandwich Cone.	37
3.4 Natural Frequencies for a Free-Free Sandwich Cone. . . . .	40
3.5 Modal Shapes for a Free-Free Sandwich Cone with $m = 1$ and $m = 2$ and Various Values of $n$ . . . . .	42
3.6 Modal Shapes for a Free-Free Sandwich Cone with $m = 3$ and $m = 4$ and Various Values of $n$ . . . . .	43
A.1 Shell Geometry . . . . .	51
G.1 Overlay Structure. . . . .	106

## SYMBOLS

$A_{im}$	Arbitrary dimensionless constants (eigenvectors) in Equation (2-11)
AI	Inertia matrix
AS	Stiffness matrix
$a_{ij}$	Constants in Equation (2-1)
$b_{ij}$	Arbitrary constants (eigenvectors) in Equation (2-2)
C	Dimensionless coefficients defined by Equation (2-9)
$C_d, C_s$	$\equiv \cos(m+k)\pi\bar{\rho}$
$E_i$	Error function or residual in Galerkin's method
$E'_x, E'_\theta$	Facing elastic modulus in x and $\theta$ directions, respectively (psi)
$\bar{E}'_x, \bar{E}'_\theta$	$\equiv E'_x/(1-\nu'_{\theta x} \nu'_{x\theta}), E'_\theta/(1-\nu'_{\theta x} \nu'_{x\theta})$ (psi)
$e_{ij}$	Strain components (in./in.)
$F_x, F_\theta$	Normal stress resultant (normal force) in x and $\theta$ directions, respectively (lb./in.)
$F_{x\theta}$	Shear stress resultant in x- $\theta$ plane (lb./in.)
f	Natural frequency (hertz)
f(x)	Arbitrary function of x
$G_{zx}, G_{\theta z}$	Core shear modulus in z-x and $\theta$ -z planes, respectively (psi)
$G'_{zx}, G'_{\theta z}, G'_{x\theta}$	Facing shear modulus in z-x, $\theta$ -z, and x- $\theta$ planes, respectively (psi)
g(x)	Arbitrary function of x
h	Core half-thickness (in.)

$h_i$	Scale factors in Equation (A-2). (Dimensionless)
J	Mass moment of inertia of core about core middle surface per unit surface area. (lb.-sec <sup>2</sup> /in.)
J'	Mass moment of inertia of one facing about core middle surface per unit surface area (lb.-sec <sup>2</sup> /in.)
$K_x, K_\theta$	Core transverse shear coefficient in z-x and $\theta$ -z planes, respectively (Dimensionless)
$K'_x, K'_\theta$	Facing transverse shear coefficient in z-x and $\theta$ -z planes, respectively (Dimensionless)
L	Shell slant length (in.)
$M_i$	Upper summation limit in assumed mode series
$M_x$	Bending moment (in.-lb./in.)
$M_{x\theta}$	Twisting moment (in.-lb./in.)
m	Meridional mode number; summation index
$\bar{m}$	Composite shell mass per unit surface area (lb.-sec <sup>2</sup> /in. <sup>3</sup> )
n	Number of circumferential full-waves
$Q_x, Q_\theta$	Transverse shear stress resultant in z-x and $\theta$ -z planes, respectively (lb./in.)
R	$\equiv \bar{R}_o + \epsilon \sin \alpha$ (Dimensionless)
$R_o$	Radius of the middle surface at the small end of the shell (in.)
$\bar{R}_o$	$\equiv R_o/L$ (Dimensionless)
r	$\equiv R_o + x \sin \alpha + z \cos \alpha$ (in.) See Equation (A-6)
s	A distance in a general curvilinear coordinate system (in.)
$S_d, S_s$	$\equiv \sin (m\bar{r}k)\pi\bar{\rho}$
T	Total kinetic energy (in.-lb.)
t	Half-thickness of one facing (in.); time (sec.)
U(x)	Normal mode form of u (in.)

$\bar{U}$	$\equiv U/L$ (Dimensionless)
$u$	Middle surface displacement in x-direction (in.); dummy variable in solution of clamped-clamped IR122
$u_x, u_\theta, u_z$	General displacements in x, $\theta$ , z directions, respectively (in.)
$V$	Normal mode form of v (in.); volume (in. <sup>3</sup> )
$V^c, V^f$	Strain energy of core and facing, respectively (in.-lb.)
$\bar{V}$	$\equiv V/L$ (Dimensionless)
$v$	Middle-surface displacement in $\theta$ -direction (in.); dummy variable in solution of clamped-clamped IR122
$W$	Normal mode form of w (in.)
$\bar{W}$	$\equiv W/L$ (Dimensionless)
$W_d, W_s$	Dummy variables in the solution of clamped-clamped IR122
$w$	Middle surface displacement in z-direction (in.)
$x$	Meridional coordinate (see Figure A.1)
$x_1, x_2$	Variables used in Equation (2-1)
$z$	Thickness coordinate (see Figure A.1)
$\alpha$	Cone semi-vertex angle (see Figure A.1); general curvilinear coordinate in Equation (A-1)
$\beta$	General curvilinear coordinate in Equation (A-1)
$\gamma$	General curvilinear coordinate in Equation (A-1)
$\epsilon$	$\equiv x/L$ (Dimensionless)
$\zeta$	$\equiv (R_o + x \sin \alpha)^{-1}$ (in. <sup>-1</sup> )
$\eta_i$	Constants defined by Equations (A-20)
$\theta$	Angular circumferential coordinate
$\lambda$	Square root of the eigenvalue of Equation (2-1)
$\nu'_{\theta x}, \nu'_{x\theta}$	Major and minor Poisson's ratios, respectively (Dimensionless)

$\rho$	Density of core material (lb.-sec. <sup>2</sup> /in. <sup>4</sup> )
$\rho'$	Density of facing material (lb.-sec. <sup>2</sup> /in. <sup>4</sup> )
$\bar{\rho}$	$\equiv \bar{R}_o / \sin \alpha$ (Dimensionless)
$\sigma_{ij}$	Stress (psi)
$\varphi_{ij}$	Assumed mode functions
$\psi_x, \psi'_x$	Angle of rotation in the meridional direction of the normal to the middle surface for the core and facing, respectively (radians)
$\bar{\psi}_x, \bar{\psi}'_x$	Normal mode form of $\psi_x$ and $\psi'_x$ , respectively (radians)
$\psi_\theta, \psi'_\theta$	Angle of rotation in the circumferential direction of the normal to the middle surface for the core and facings, respectively (radians)
$\bar{\psi}_\theta, \bar{\psi}'_\theta$	Normal mode form of $\psi_\theta$ and $\psi'_\theta$ , respectively (radians)
$\omega$	Circular frequency (radians/sec.)

#### Superscripts

c	Refers to core
i	Refers to inner facing
o	Refers to outer facing

FREE VIBRATIONS OF ORTHOTROPIC CONICAL SANDWICH SHELLS  
WITH VARIOUS BOUNDARY CONDITIONS

CHAPTER I

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INTRODUCTION

1.1 Survey of Shell Dynamics

The extensive use of shell structures in aircraft and space vehicles has provided a great impetus to shell dynamics research. This fact is verified by the large amount of literature available on the subject. Most of the investigators have considered thin, elastic shells of homogeneous, isotropic materials. Although sandwich construction is being used more widely in shell configurations for aerospace structures, only a very limited number of analyses involving such structures has been attempted.

Apparently the first such analysis was performed by Yu [1] for the free vibrations of a simply supported sandwich cylinder. Other work on sandwich cylinders includes that of Chu [2], [3] on wave propagation and large-amplitude vibration, Bieniek and Freudenthal [4] on harmonic forced vibration, Yu [5] and Jones and Salerno [6] on structural damping, and Greenspon [7] on the effect of initial stress.

Analyses for other configurations include those by Mead and Pretlove [8] and Jacobson and Wenner [9] for cylindrically curved

panels, Tasi [10] and Koplik and Yu [11], [12], [13] for shallow spherical-shell caps and Suvernev [14], as reported in [15], for conical frustums.

None of the above-mentioned analyses considered orthotropic facings, as exemplified by fiber-reinforced composites. However, some did allow for the commonly used hexagonal-cell honeycomb core by considering the simpler effect of an orthotropic core. The first sandwich-shell vibration analyses to consider orthotropic facings and core were done independently by Azar [16], Vasitsyna [17] and Baker and Herrmann [18]. Azar treated axisymmetric free vibrations of freely supported arbitrary open-ended shells of revolution, such as conical and paraboloidal shell frusta but excluding cylinders. Vasitsyna analyzed free vibrations of simply supported circular cylinders, while Baker and Herrmann considered the same case with the addition of a general state of initial stress. Later, Bacon and Bert [19] extended Azar's work to include unsymmetric modes.

Most of the above analyses used simply supported edges, while a few considered clamped edges. None of them considered free edges. In fact, until very recently, the most difficult case that had been analyzed for free edges was that of a homogeneous, isotropic, conical shell. Hu [20] formulated his analysis to include both membrane and bending effects, with solutions by Galerkin's method. However, the only numerical results which he published for free-free boundary conditions included only membrane effects and they were not compared with any experimental results. An analysis by Hu, et al [21] considered the conical shell to be inextensional, i.e., the membrane strains were

identically zero, and used the Rayleigh-Ritz technique. Their results compared favorably with their previous experimental results [22] for low values of circumferential wave number,  $n$ . Sewall's analysis, mentioned in a report by Mixson [23], was said to have been solved by a Rayleigh-Ritz technique, and his results, for the lowest unsymmetric mode only, agreed quite well with Mixson's experimental values for homogeneous, isotropic, conical shells. The analysis of Krause [24], for the same type of shell, included both membrane and bending effects, and used a modified Galerkin method in which it was not necessary to satisfy the force and moment boundary conditions. However, Krause's results did not appear to agree as well with the experimental values of [22] as did those obtained by the much simpler analysis of [21]. The case of a sandwich conical frustum with orthotropic facings, perfectly rigid core, and free edges was analyzed by Bert, et al [25] by using a simple inextensional theory and the Rayleigh-Ritz method. Their analysis agreed well with their experiments on a free-free, orthotropic, sandwich shell.

### 1.2 Research Objectives

The purpose of this research is to develop a general analysis, with accompanying computer program, with the following capabilities and characteristics:

1. Shell Geometry - conical frustum with cylinder as a special case.
2. Material - linearly elastic; either isotropic or orthotropic.
3. Facing Flexibilities - all components of extensional,

flexural, and shear strain, but neglecting coupling between extensional and flexural effects (i.e., Love first approximation shell theory plus shear).

4. Core Flexibilities - transverse shear only, as is standard for a sandwich core.
5. Inertia effects - all components of translational and rotatory inertia.
6. Type of mode - axisymmetric or unsymmetric.
7. Boundary conditions - arbitrary - to be specified as input functions for different sets of end conditions, including clamped-clamped, freely supported and free-free.
8. Kind of construction - symmetrical sandwich (symmetric about the middle surface) with homogeneous as special case.

## CHAPTER II

### FORMULATION OF THE THEORY

#### 2.1 Method of Analysis

First, expressions for the kinetic and potential energies of a symmetrical sandwich conical shell with orthotropic facings and core are derived from basic principles. This derivation is presented in Appendix A. Next, in Appendix B, Hamilton's principle is employed to derive the differential equations of motion and the boundary conditions. Galerkin's method is then applied to the equations of motion. The result of this operation is a set of simultaneous linear algebraic equations in the form of a standard eigenvalue problem. The eigenvalue problem is then solved with the aid of a digital computer.

#### 2.2 Hypotheses

All of the following assumptions will be implicit in the analysis:

1. The core is capable of resisting transverse shear, but not bending, extension, or in-plane shear.
2. The facings resist extension, bending, and transverse and in-plane shear.
3. The facings are identical, so that the sandwich is of symmetrical construction.

4. Both the core and the facings are linearly elastic and can be orthotropic.
5. The facings and core furnish both translational and rotatory inertia effects.
6. The shell thickness is small compared to the smallest radius of curvature of the shell, so that  $z/r$  may be neglected when compared to unity.
7. All deflections are small, so that the strain-displacement relations can be linearized.
8. Lines which are straight and normal to the middle surface before deformation remain straight during deformation, but do not necessarily remain normal to the middle surface.
9. The facing rotation,  $\psi'_x$ , is assumed to be identical in the inner and outer facings, in view of hypothesis (6). The same assumption also applies to  $\psi'_\theta$ .
10. All material damping, thermal, and initial-stress effects, as well as interactions with surrounding fluid, are neglected.

### 2.3 Application of Galerkin's Method

The Galerkin method is an approximate (assumed-mode) method for the solution of boundary-value problems. To apply it, one begins with the equations of motion. Solutions are assumed for the unknown variables of the problem. These assumed solutions must satisfy the boundary conditions but do not necessarily satisfy the equations of

motion. This means that the insertion of the assumed solution into each of the motion equations results in a non-zero "error function" rather than zero, which would be the result if the exact solution were known. In order to minimize the error, each of the error functions is required to be orthogonal to the assumed functions. This orthogonalization process gives rise to a set of simultaneous, linear, homogeneous equations with the vibrational frequency as the eigenvalue.

As an example in the use of Galerkin's method, consider the simple set of equations

$$\begin{aligned}(a_{11} - \lambda^2)x_1 + a_{12}x_2 &= 0 \\ a_{21}x_1 + (a_{22} - \lambda^2)x_2 &= 0\end{aligned}\tag{2-1}$$

Now assume

$$\begin{aligned}x_1 &= \sum_{m=1}^2 b_{1m}\varphi_{1m} = b_{11}\varphi_{11} + b_{12}\varphi_{12} \\ x_2 &= \sum_{m=1}^2 b_{2m}\varphi_{2m} = b_{21}\varphi_{21} + b_{22}\varphi_{22}\end{aligned}\tag{2-2}$$

where the  $\varphi$  functions satisfy the boundary conditions independent of the values of the  $b$ 's. Using Equations (2-2) in Equations (2-1) results in

$$\begin{aligned}E_1 &= (a_{11} - \lambda^2)(b_{11}\varphi_{11} + b_{12}\varphi_{12}) + a_{12}(b_{21}\varphi_{21} + b_{22}\varphi_{22}) \\ E_2 &= a_{21}(b_{11}\varphi_{11} + b_{12}\varphi_{12}) + (a_{22} - \lambda^2)(b_{21}\varphi_{21} + b_{22}\varphi_{22})\end{aligned}\tag{2-3}$$

where  $E_1$  and  $E_2$  are non-zero since Equations (2-2) do not represent the exact solutions for  $x_1$  and  $x_2$ . In order to minimize  $E_1$  and  $E_2$ , Galerkin's method requires that

$$\int_V E_1 \varphi_{1k} d(\text{vol.}) = 0$$

$$\int_V E_2 \varphi_{2k} d(\text{vol.}) = 0 \quad (2-4)$$

for  $k=1, 2$ .

The functions  $\varphi_{1k}$  and  $\varphi_{2k}$  were chosen for the orthogonalization to insure that the  $\lambda^2$  terms would not vanish. The first of the four Equations (2-4) is

$$\begin{aligned} \int_V E_1 \varphi_{11}^2 dV &= a_{11} b_{11} \int_V \varphi_{11}^2 dV + a_{11} b_{12} \int_V \varphi_{12} \varphi_{11} dV \\ &+ a_{12} b_{21} \int_V \varphi_{21} \varphi_{11} dV + a_{12} b_{22} \int_V \varphi_{22} \varphi_{11} dV \\ &- \lambda^2 b_{11} \int_V \varphi_{11}^2 dV - \lambda^2 b_{12} \int_V \varphi_{12} \varphi_{11} dV = 0 \quad (2-5) \end{aligned}$$

The equations would be put in matrix form as follows:

$$\left[ \begin{array}{cccc} a_{11} \int_V \varphi_{11}^2 dV & a_{11} \int_V \varphi_{12} \varphi_{11} dV & a_{12} \int_V \varphi_{21} \varphi_{11} dV & a_{12} \int_V \varphi_{22} \varphi_{11} dV \\ \vdots & \vdots & \vdots & \vdots \end{array} \right] - \lambda^2 \left[ \begin{array}{cccc} \int_V \varphi_{11}^2 dV & \int_V \varphi_{12} \varphi_{11} dV & 0 & 0 \\ \vdots & \vdots & \vdots & \vdots \end{array} \right] \begin{bmatrix} b_{11} \\ b_{12} \\ b_{21} \\ b_{22} \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \\ 0 \\ 0 \end{bmatrix} \quad (2-6)$$

As noted by Yu and Lai [26] and Singer [27], Galerkin's method is equivalent to the Rayleigh-Ritz method, when applied correctly. The main restriction on Galerkin's method is making certain that the procedure is applied to the coupled equations of motion which arise directly from equilibrium considerations or from Hamilton's principle. This means that differentiations may not be performed to uncouple the

equations of motion. The equivalence of the Galerkin and Rayleigh-Ritz methods refers to the fact that both methods use a minimum principle, and should not imply that identical results are obtained from both methods. In some special cases, identical stiffness and inertia matrices, and hence, identical results, are obtained from both methods, but in general, Galerkin's method gives rise to a non-symmetric stiffness matrix, while the Rayleigh-Ritz method always gives rise to a symmetric stiffness matrix.

Now, with this introduction to Galerkin's method, and the results of Appendices A and B, the solution to the conical sandwich shell vibration problem may commence.

In order to remove the time dependence and the circumferential dependence, the following forms are assumed:

$$\begin{aligned}
 u(x, \theta, t) &= U(x) \cos n\theta \sin \omega t \\
 v(x, \theta, t) &= V(x) \sin n\theta \sin \omega t \\
 w(x, \theta, t) &= W(x) \cos n\theta \sin \omega t \\
 \psi_{\theta}'(x, \theta, t) &= \bar{\psi}_{\theta}'(x) \sin n\theta \sin \omega t \\
 \psi_{x}'(x, \theta, t) &= \bar{\psi}_{x}'(x) \cos n\theta \sin \omega t \\
 \psi_{\theta}(x, \theta, t) &= \bar{\psi}_{\theta}(x) \sin n\theta \sin \omega t \\
 \psi_{x}(x, \theta, t) &= \bar{\psi}_{x}(x) \cos n\theta \sin \omega t
 \end{aligned} \tag{2-7}$$

Substitution of Equations (2-7) into Equations (B-5) gives

$$\begin{aligned}
 &-2\eta_1(U_{,xx} + \zeta \sin \alpha U_{,x}) + 2\eta_{12}n^2\zeta^2U + 2\eta_2 \sin^2 \alpha \zeta^2U \\
 &+ 2n \sin \alpha (\eta_2 + \eta_{12})\zeta^2V - n(2\eta_{12} + \eta_3)\zeta V_{,x} \\
 &+ 2\eta_2 \sin \alpha \cos \alpha \zeta^2W - \eta_3 \cos \alpha \zeta W_{,x} - \bar{m}\omega^2U = 0
 \end{aligned}$$

$$\begin{aligned}
 & - 2\eta^2 \omega^2 \psi^x = 0 \\
 & - \eta^9 (\psi^x)_{xx} + \zeta \sin \alpha (\psi^x)_{11} + \eta^2 \sin^2 \alpha (\psi^x)_{13} + \zeta^2 \psi^x \\
 & + 2\eta^2 \zeta^2 \omega^2 \psi^x - \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & + 2\eta^2 \omega^2 \psi^x - \eta^2 \omega^2 \psi^x + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & + 2\eta^2 \omega^2 \psi^x - \eta^2 \omega^2 \psi^x + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x
 \end{aligned}$$

$$\begin{aligned}
 & 0 = \theta^2 \psi^x - 2\eta^2 \omega^2 \psi^x \\
 & + \eta^2 \omega^2 \psi^x \\
 & + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & - 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x
 \end{aligned}$$

$$\begin{aligned}
 & + \zeta \sin \alpha (\psi^x)_{11} - \eta^2 \omega^2 \psi^x = 0 \\
 & - 2\eta^2 \omega^2 \psi^x + \zeta \sin \alpha (\psi^x)_{11} - 2\eta^2 \omega^2 \psi^x + \zeta^2 \psi^x \\
 & + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & + \eta^2 \omega^2 \psi^x \\
 & \eta^2 \cos \alpha (\psi^x)_{11} + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x
 \end{aligned}$$

$$\begin{aligned}
 & - 2\eta^2 \omega^2 \psi^x + \zeta \sin \alpha (\psi^x)_{11} - 2\eta^2 \omega^2 \psi^x = 0 \\
 & + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & + \zeta \sin \alpha (\psi^x)_{11} + 2\eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x \\
 & \eta^2 \cos \alpha (\psi^x)_{11} + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x + \eta^2 \omega^2 \psi^x
 \end{aligned}$$

$$\begin{aligned}
& -2\eta_{16} \cos \alpha \zeta V - 2n\eta_{16} \zeta W - \eta_{13} (\bar{\psi}'_{\theta,xx} + \zeta \sin \alpha \bar{\psi}'_{\theta,x}) \\
& + (2\eta_5 h t \cos^2 \alpha + \eta_{13} \sin^2 \alpha) \zeta^2 \bar{\psi}'_{\theta} + n^2 \eta_{11} \zeta^2 \bar{\psi}'_{\theta} \\
& + n(\eta_{10} + \eta_{13}) \zeta \bar{\psi}'_{x,x} + n(\eta_{11} + \eta_{13}) \sin \alpha \zeta^2 \bar{\psi}'_x \\
& - 2\eta_{12} h^2 (\bar{\psi}_{\theta,xx} + \zeta \sin \alpha \bar{\psi}_{\theta,x}) + 2(\eta_5 h^2 \cos^2 \alpha \zeta^2 \\
& + \eta_{12} h^2 \sin^2 \alpha \zeta^2 + \eta_{16}) \bar{\psi}_{\theta} + 2n^2 \eta_2 h^2 \zeta^2 \bar{\psi}_{\theta} + n(\eta_3 h^2 \\
& + 2\eta_{12} h^2) \zeta \bar{\psi}_{x,x} + 2nh^2 \sin \alpha (\eta_2 + \eta_{12}) \zeta^2 \bar{\psi}_x - J\omega^2 \bar{\psi}_{\theta} = 0
\end{aligned}$$

$$\begin{aligned}
& + 2\eta_{15} W_{,x} - n(\eta_{10} + \eta_{13}) \zeta \bar{\psi}'_{\theta,x} + n(\eta_{11} + \eta_{13}) \sin \alpha \zeta^2 \bar{\psi}'_{\theta} \\
& - \eta_9 (\bar{\psi}'_{x,xx} + \zeta \sin \alpha \bar{\psi}'_{x,x}) + (\eta_{11} \sin^2 \alpha + n^2 \eta_{13}) \zeta^2 \bar{\psi}'_x \\
& - nh^2 (\eta_3 + 2\eta_{12}) \zeta \bar{\psi}_{\theta,x} + 2nh^2 \sin \alpha (\eta_2 + \eta_{12}) \zeta^2 \bar{\psi}_{\theta} \\
& - 2\eta_1 h^2 (\bar{\psi}_{x,xx} + \zeta \sin \alpha \bar{\psi}_{x,x}) + 2(\eta_2 h^2 \sin^2 \alpha \zeta^2 \\
& + \eta_{15}) \bar{\psi}_x + 2n^2 \eta_{12} h^2 \zeta^2 \bar{\psi}_x - J\omega^2 \bar{\psi}_x = 0
\end{aligned} \tag{2-8}$$

Now, define:

$$e \equiv x/L$$

$$\bar{U} \equiv U/L$$

$$\bar{V} \equiv V/L$$

$$\bar{W} \equiv W/L$$

$$R \equiv (\zeta L)^{-1} = (R_0 + x \sin \alpha)/L = \bar{R}_0$$

$$+ e \sin \alpha$$

$$C_{11} = 2(\eta_{12} n^2 + \eta_2 \sin^2 \alpha)/E'_x L$$

$$C_{\bar{m}} = 4\pi^2 \bar{m} L / E'_x$$

$$C_{111} = -2\eta_1 \sin \alpha / E'_x L$$

$$\begin{aligned}
C_{112} &= -2\eta_1/E'_x L \\
C_{12} &= 2n \sin \alpha (\eta_2 + \eta_{12})/E'_x L \\
C_{121} &= -n(2\eta_{12} + \eta_3)/E'_x L \\
C_{13} &= 2\eta_2 \sin \alpha \cos \alpha/E'_x L \\
C_{131} &= -\eta_3 \cos \alpha/E'_x L \\
C_{22} &= 2[\cos^2 \alpha (\eta_5 + \eta_{16}) + \sin^2 \alpha \eta_{12} + n^2 \eta_2]/E'_x L \\
C_{221} &= -2\eta_{12} \sin \alpha/E'_x L \\
C_{222} &= -2\eta_{12}/E'_x L \\
C_{23} &= 2n \cos \alpha (\eta_2 + \eta_5 + \eta_{16})/E'_x L \\
C_{24} &= -2\eta_5 \cos \alpha/E'_x L \\
C_{26} &= -2\eta_{16} \cos \alpha/E'_x L \\
C_{33} &= 2[n^2 (\eta_5 + \eta_{16}) + \cos^2 \alpha \eta_2]/E'_x L \\
C_{331} &= -2 \sin \alpha (\eta_4 + \eta_{15})/E'_x L \\
C_{332} &= -2(\eta_4 + \eta_{15})/E'_x L \\
C_{34} &= -2\eta_5 n/E'_x L \\
C_{35} &= -2\eta_4 \sin \alpha/E'_x L \\
C_{351} &= -2\eta_4/E'_x L \\
C_{36} &= -2\eta_{16} n/E'_x L \\
C_{37} &= -2\eta_{15} \sin \alpha/E'_x L \\
C_{371} &= -2\eta_{15}/E'_x L \\
C_{44} &= 2(n^2 \eta_7 + \sin^2 \alpha \eta_{14})/E'_x L^3
\end{aligned}$$

$$\begin{aligned}
C_{440} &= 2\eta_5/E'_x L \\
C_{J'} &= 8\pi^2 J'/E'_x L \\
C_{441} &= 2\eta_{14} \sin \alpha / E'_x L^3 \\
C_{442} &= -2\eta_{14}/E'_x L^3 \\
C_{45} &= 2n \sin \alpha (\eta_7 + \eta_{14})/E'_x L^3 \\
C_{451} &= -n(\eta_8 + 2\eta_{14})/E'_x L^3 \\
C_{46} &= (n^2 \eta_{11} + 2\eta_5 h t \cos^2 \alpha + \eta_{13} \sin^2 \alpha)/E'_x L^3 \\
C_{461} &= -\eta_{13} \sin \alpha / E'_x L^3 \\
C_{462} &= -\eta_{13}/E'_x L^3 \\
C_{47} &= n \sin \alpha (\eta_{11} + \eta_{13})/E'_x L^3 \\
C_{471} &= -n(\eta_{10} + \eta_{13})/E'_x L^3 \\
C_{55} &= 2(\eta_7 \sin^2 \alpha + n^2 \eta_{14})/E'_x L^3 \\
C_{550} &= 2\eta_4/E'_x L \\
C_{551} &= -2\eta_6 \sin \alpha / E'_x L^3 \\
C_{552} &= -2\eta_6/E'_x L^3 \\
C_{56} &= n \sin \alpha (\eta_{11} + \eta_{13})/E'_x L^3 \\
C_{561} &= -n(\eta_{10} + \eta_{13})/E'_x L^3 \\
C_{57} &= (\eta_{11} \sin^2 \alpha + n^2 \eta_{13})/E'_x L^3 \\
C_{571} &= -\eta_9 \sin \alpha / E'_x L^3 \\
C_{572} &= -\eta_9/E'_x L^3 \\
C_{66} &= 2h^2(\eta_5 \cos^2 \alpha + \eta_{12} \sin^2 \alpha + n^2 \eta_2)/E'_x L^3
\end{aligned}$$

$$\begin{aligned}
C_{660} &= 2\eta_{16}/E'_x L \\
C_{661} &= -2h^2 \eta_{12} \sin \alpha / E'_x L^3 \\
C_{662} &= -2h^2 \eta_{12} / E'_x L^3 \\
C_{67} &= 2nh^2 \sin \alpha (\eta_2 + \eta_{12}) / E'_x L^3 \\
C_{671} &= -nh^2 (\eta_3 + 2\eta_{12}) / E'_x L^3 \\
C_{77} &= 2h^2 (\eta_2 \sin^2 \alpha + n^2 \eta_{12}) / E'_x L^3 \\
C_{770} &= 2\eta_{15} / E'_x L \\
C_{771} &= -2\eta_1 h^2 \sin \alpha / E'_x L^3 \\
C_{772} &= -2\eta_1 h^2 / E'_x L^3 \\
C_J &= 4\pi^2 J / E'_x L \quad (2-9)
\end{aligned}$$

The use of Equations (2-9) in Equations (2-8) results in the following non-dimensional equations of motion.

$$\begin{aligned}
C_{11} R^{-2} \bar{U} + C_{111} R^{-1} \bar{U}_{,\epsilon} + C_{112} \bar{U}_{,\epsilon\epsilon} + C_{12} R^{-2} \bar{V} + C_{121} R^{-1} \bar{V}_{,\epsilon} + C_{13} R^{-2} \bar{W} \\
+ C_{131} R^{-1} \bar{W}_{,\epsilon} - C_m f^2 \bar{U} = 0
\end{aligned}$$

$$\begin{aligned}
C_{12} R^{-2} \bar{U} - C_{121} R^{-1} \bar{U}_{,\epsilon} + C_{22} R^{-2} \bar{V} + C_{221} R^{-1} \bar{V}_{,\epsilon} + C_{222} \bar{V}_{,\epsilon\epsilon} + C_{23} R^{-2} \bar{W} \\
+ C_{24} R^{-1} \bar{V}'_{\theta} + C_{26} R^{-1} \bar{V}'_{\theta} - C_m f^2 \bar{V} = 0
\end{aligned}$$

$$\begin{aligned}
C_{13} R^{-2} \bar{U} - C_{131} R^{-1} \bar{U}_{,\epsilon} + C_{23} R^{-2} \bar{V} + C_{33} R^{-2} \bar{W} + C_{331} R^{-1} \bar{W}_{,\epsilon} + C_{332} \bar{W}_{,\epsilon\epsilon} \\
+ C_{34} R^{-1} \bar{V}'_{\theta} + C_{35} R^{-1} \bar{V}'_{\theta} + C_{351} \bar{V}'_{\theta,\epsilon} + C_{36} R^{-1} \bar{V}'_{\theta} + C_{37} R^{-1} \bar{V}'_{\theta} \\
+ C_{371} \bar{V}'_{\theta,\epsilon} - C_m f^2 \bar{W} = 0
\end{aligned}$$

$$\begin{aligned}
& C_{24}R^{-1}\bar{V} + C_{34}R^{-1}\bar{W} + C_{44}R^{-2}\bar{\Psi}'_{\theta} + C_{440}\bar{\Psi}'_{\theta} + C_{441}R^{-1}\bar{\Psi}'_{\theta,\epsilon} + C_{442}\bar{\Psi}'_{\theta,\epsilon\epsilon} \\
& + C_{45}R^{-2}\bar{\Psi}'_{x} - C_{451}R^{-1}\bar{\Psi}'_{x,\epsilon} + C_{46}R^{-2}\bar{\Psi}'_{\theta} + C_{461}R^{-1}\bar{\Psi}'_{\theta,\epsilon} \\
& + C_{462}\bar{\Psi}'_{\theta,\epsilon\epsilon} + C_{47}R^{-2}\bar{\Psi}'_{x} - C_{471}R^{-1}\bar{\Psi}'_{x,\epsilon} - C_J f^2 \bar{\Psi}'_{\theta} = 0 \\
\\
& -C_{351}\bar{W}_{,\epsilon} + C_{45}R^{-2}\bar{\Psi}'_{\theta} + C_{451}R^{-1}\bar{\Psi}'_{\theta,\epsilon} + C_{55}R^{-2}\bar{\Psi}'_{x} + C_{550}\bar{\Psi}'_{x} \\
& + C_{551}R^{-1}\bar{\Psi}'_{x,\epsilon} + C_{552}\bar{\Psi}'_{x,\epsilon\epsilon} + C_{56}R^{-2}\bar{\Psi}'_{\theta} + C_{561}R^{-1}\bar{\Psi}'_{\theta,\epsilon} \\
& + C_{57}R^{-2}\bar{\Psi}'_{x} + C_{571}R^{-1}\bar{\Psi}'_{x,\epsilon} + C_{572}\bar{\Psi}'_{x,\epsilon\epsilon} - C_J f^2 \bar{\Psi}'_{x} = 0 \\
\\
& C_{26}R^{-1}\bar{V} + C_{36}R^{-1}\bar{W} + C_{46}R^{-2}\bar{\Psi}'_{\theta} + C_{461}R^{-1}\bar{\Psi}'_{\theta,\epsilon} + C_{462}\bar{\Psi}'_{\theta,\epsilon\epsilon} \\
& + C_{56}R^{-2}\bar{\Psi}'_{x} - C_{561}R^{-1}\bar{\Psi}'_{x,\epsilon} + C_{66}R^{-2}\bar{\Psi}'_{\theta} + C_{660}\bar{\Psi}'_{\theta} \\
& + C_{661}R^{-1}\bar{\Psi}'_{\theta,\epsilon} + C_{662}\bar{\Psi}'_{\theta,\epsilon\epsilon} + C_{67}R^{-2}\bar{\Psi}'_{x} - C_{671}R^{-1}\bar{\Psi}'_{x,\epsilon} \\
& - C_J f^2 \bar{\Psi}'_{\theta} = 0 \\
\\
& -C_{371}\bar{W}_{,\epsilon} + C_{47}R^{-2}\bar{\Psi}'_{\theta} + C_{471}R^{-1}\bar{\Psi}'_{\theta,\epsilon} + C_{57}R^{-2}\bar{\Psi}'_{x} + C_{571}R^{-1}\bar{\Psi}'_{x,\epsilon} \\
& + C_{572}\bar{\Psi}'_{x,\epsilon\epsilon} + C_{67}R^{-2}\bar{\Psi}'_{\theta} + C_{671}R^{-1}\bar{\Psi}'_{\theta,\epsilon} + C_{77}R^{-2}\bar{\Psi}'_{x} \\
& + C_{770}\bar{\Psi}'_{x} + C_{771}R^{-1}\bar{\Psi}'_{x,\epsilon} + C_{772}\bar{\Psi}'_{x,\epsilon\epsilon} - C_J f^2 \bar{\Psi}'_{x} = 0 \quad (2-10)
\end{aligned}$$

A solution whose form is dependent on the boundary conditions must now be assumed. This step is accomplished symbolically by letting

$$\bar{U} = \sum_m A_{1m} \phi_{1m}(\epsilon)$$

$$\bar{V} = \sum_m A_{2m} \phi_{2m}(\epsilon)$$

$$\begin{aligned}
\bar{w} &= \sum_m A_{3m} \varphi_{3m}(\epsilon) \\
\bar{\psi}'_{\theta} &= \sum_m A_{4m} \varphi_{4m}(\epsilon) \\
\bar{\psi}'_x &= \sum_m A_{5m} \varphi_{5m}(\epsilon) \\
\bar{\psi}_{\theta} &= \sum_m A_{6m} \varphi_{6m}(\epsilon) \\
\bar{\psi}_x &= \sum_m A_{7m} \varphi_{7m}(\epsilon)
\end{aligned} \tag{2-11}$$

where  $\varphi_{1m}, \varphi_{2m}, \dots, \varphi_{7m}$  are functions satisfying the appropriate end conditions. Selection of these functions will be discussed in Section 2.4.

Thus, substitution of Equations (2-11) into Equations (2-10) gives a set of expressions for the error functions.

$$\begin{aligned}
E_1 = \sum_m \left\{ \right. & [ (C_{11} R^{-2} - f^2 C_m^-) \varphi_{1m} + C_{111} R^{-1} \varphi_{1m,\epsilon} + C_{112} \varphi_{1m,\epsilon\epsilon} ] A_{1m} \\
& + [ C_{12} R^{-2} \varphi_{2m} + C_{121} R^{-1} \varphi_{2m,\epsilon} ] A_{2m} + [ C_{13} R^{-2} \varphi_{3m} \\
& \left. + C_{131} R^{-1} \varphi_{3m,\epsilon} ] A_{3m} \right\}
\end{aligned}$$

$$\begin{aligned}
E_2 = \sum_m \left\{ \right. & [ C_{12} R^{-2} \varphi_{1m} - C_{121} R^{-1} \varphi_{1m,\epsilon} ] A_{1m} + [ (C_{22} R^{-2} - f^2 C_m^-) \varphi_{2m} \\
& + C_{221} R^{-1} \varphi_{2m,\epsilon} + C_{222} \varphi_{2m,\epsilon\epsilon} ] A_{2m} + [ C_{23} R^{-2} \varphi_{3m} ] A_{3m} \\
& \left. + [ C_{24} R^{-1} \varphi_{4m} ] A_{4m} + [ C_{26} R^{-1} \varphi_{6m} ] A_{6m} \right\}
\end{aligned}$$

$$\begin{aligned}
E_3 = \sum_m \left\{ \right. & [ C_{13} R^{-2} \varphi_{1m} - C_{131} R^{-1} \varphi_{1m,\epsilon} ] A_{1m} + [ C_{23} R^{-2} \varphi_{2m} ] A_{2m} \\
& + [ (C_{33} R^{-2} - f^2 C_m^-) \varphi_{3m} + C_{331} R^{-1} \varphi_{3m,\epsilon} + C_{332} \varphi_{3m,\epsilon\epsilon} ] A_{3m} \\
& + [ C_{34} R^{-1} \varphi_{4m} ] A_{4m} + [ C_{35} R^{-1} \varphi_{5m} + C_{351} \varphi_{5m,\epsilon} ] A_{5m} \\
& \left. + [ C_{36} R^{-1} \varphi_{6m} ] A_{6m} + [ C_{37} R^{-1} \varphi_{7m} + C_{371} \varphi_{7m,\epsilon} ] A_{7m} \right\}
\end{aligned}$$

$$\begin{aligned}
E_4 &= \sum_m \left\{ [C_{24} R^{-1} \varphi_{2m}] A_{2m} + [C_{34} R^{-1} \varphi_{3m}] A_{3m} + [(C_{44} R^{-2} + C_{440} \right. \\
&\quad - f^2 C_{J'}) \varphi_{4m} + C_{441} R^{-1} \varphi_{4m,e} + C_{442} \varphi_{4m,ee}] A_{4m} + [C_{45} R^{-2} \varphi_{5m} \\
&\quad - C_{451} R^{-1} \varphi_{5m,e}] A_{5m} + [C_{46} R^{-2} \varphi_{6m} + C_{461} R^{-1} \varphi_{6m,e} \\
&\quad \left. + C_{462} \varphi_{6m,ee}] A_{6m} + [C_{47} R^{-2} \varphi_{7m} - C_{471} R^{-1} \varphi_{7m,e}] A_{7m} \right\} \\
E_5 &= \sum_m \left\{ [-C_{351} \varphi_{3m,e}] A_{3m} + [C_{45} R^{-2} \varphi_{4m} + C_{451} R^{-1} \varphi_{4m,e}] A_{4m} \right. \\
&\quad + [(C_{55} R^{-2} + C_{550} - f^2 C_{J'}) \varphi_{5m} + C_{551} R^{-1} \varphi_{5m,e} \\
&\quad + C_{552} \varphi_{5m,ee}] A_{5m} + [C_{56} R^{-2} \varphi_{6m} + C_{561} R^{-1} \varphi_{6m,e}] A_{6m} \\
&\quad \left. + [C_{57} R^{-2} \varphi_{7m} + C_{571} R^{-1} \varphi_{7m,e} + C_{572} \varphi_{7m,ee}] A_{7m} \right\} \\
E_6 &= \sum_m \left\{ [C_{26} R^{-1} \varphi_{2m}] A_{2m} + [C_{36} R^{-1} \varphi_{3m}] A_{3m} + [C_{46} R^{-2} \varphi_{4m} \right. \\
&\quad + C_{461} R^{-1} \varphi_{4m,e} + C_{462} \varphi_{4m,ee}] A_{4m} + [C_{56} R^{-2} \varphi_{5m} \\
&\quad - C_{561} R^{-1} \varphi_{5m,e}] A_{5m} + [(C_{66} R^{-2} + C_{660} - f^2 C_J) \varphi_{6m} \\
&\quad + C_{661} R^{-1} \varphi_{6m,e} + C_{662} \varphi_{6m,ee}] A_{6m} + [C_{67} R^{-2} \varphi_{7m} \\
&\quad \left. - C_{671} R^{-1} \varphi_{7m,e}] A_{7m} \right\} \\
E_7 &= \sum_m \left\{ [-C_{371} \varphi_{3m,e}] A_{3m} + [C_{47} R^{-2} \varphi_{4m} + C_{471} R^{-1} \varphi_{4m,e}] A_{4m} \right. \\
&\quad + [C_{57} R^{-2} \varphi_{5m} + C_{571} R^{-1} \varphi_{5m,e} + C_{572} \varphi_{5m,ee}] A_{5m} \\
&\quad + [C_{67} R^{-2} \varphi_{6m} + C_{671} R^{-1} \varphi_{6m,e}] A_{6m} + [(C_{77} R^{-2} + C_{770} \\
&\quad \left. - f^2 C_J) \varphi_{7m} + C_{771} R^{-1} \varphi_{7m,e} + C_{772} \varphi_{7m,ee}] A_{7m} \right\} \tag{2-12}
\end{aligned}$$

In order for the error functions to be orthogonal to the assumed functions,

$$\int_0^1 E_1 \varphi_{1k} de = 0$$

$$\int_0^1 E_2 \varphi_{2k} d\epsilon = 0$$

$$\int_0^1 E_3 \varphi_{3k} d\epsilon = 0$$

$$\int_0^1 E_4 \varphi_{4k} d\epsilon = 0$$

$$\int_0^1 E_5 \varphi_{5k} d\epsilon = 0$$

$$\int_0^1 E_6 \varphi_{6k} d\epsilon = 0$$

$$\int_0^1 E_7 \varphi_{7k} d\epsilon = 0 \quad (2-13)$$

for each value of  $k$ , where its range is the same as that of  $m$  in Equations (2-11). Equations (2-13) may now be put in matrix form. The terms involving  $f^2$  are separated so that

$$[AS] \{y\} - f^2 [AI] \{y\} = 0 \quad (2-14)$$

The matrix  $[AS]$  is the stiffness matrix and the matrix  $[AI]$  is the inertia matrix. The matrix  $\{y\}$  is a column matrix of unknown  $A$ 's. Equation (2-14) is the familiar form of an eigenvalue problem, with  $f^2$  as the eigenvalue and  $\{y\}$  as the eigenvector. The elements of the stiffness and inertia matrices are made up of various combinations of integrals involving the assumed functions,  $\varphi$ , which, of course, depend upon the boundary conditions.

The figures on the following pages show the form of the stiffness matrix for various conditions. Figure 2.1 shows the general

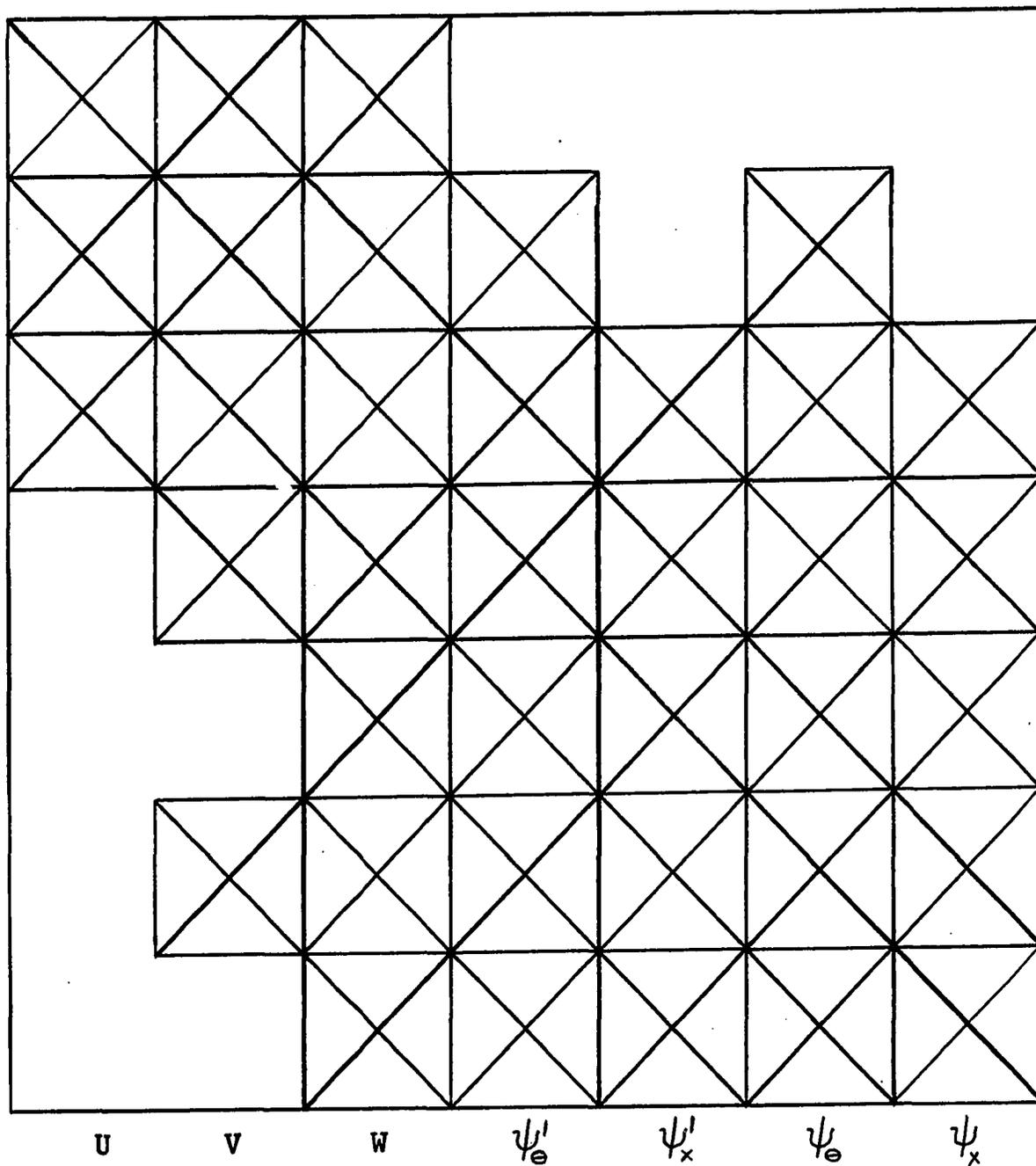


Figure 2.1 - General Form of Stiffness Matrix

form, where the cross-hatched area is populated and the rest of the matrix is zero. Figure 2.2 shows that other submatrices become zero for  $n = 0$ . Figure 2.3 shows that two rows and two columns of zeros occur for a homogeneous shell.

#### 2.4 Boundary Conditions

The full set of boundary conditions arises as a by-product of applying Hamilton's principle to find the equations of motion. This is very advantageous since the boundary conditions derived in this manner are certain to be compatible with the equations of motion, within the framework of the assumptions implied in the shell theory used. Of course, for a given practical boundary condition, one must choose which of the parts of the full set to use. In this study, three boundary conditions are to be investigated: freely supported, clamped-clamped and free-free.

The set of boundary conditions from Equations (B-12), after applying Equations (2-7), is written as follows:

$$\text{Either } U = 0,$$

$$\text{or } F_x = 2\eta_1 U_{,x} + \eta_3 (\sin \alpha \zeta U + n\zeta V + \cos \alpha \zeta W) = 0 \quad (2-15)$$

$$\text{Either } V = 0,$$

$$\text{or } F_{x\theta} = 2\eta_{12} (-n\zeta U + V_{,x} - \sin \alpha \zeta V) = 0 \quad (2-16)$$

$$\text{Either } W = 0,$$

$$\text{or } Q_x = 2(\eta_4 + \eta_{15})W_{,x} + 2\eta_4 \bar{\psi}'_x + 2\eta_{15} \bar{\psi}_x = 0 \quad (2-17)$$

$$\text{Either } \bar{\psi}'_\theta = \bar{\psi}_\theta = 0,$$

$$\text{or } M_{x\theta} = (2\eta_{14} + \eta_{13})(\bar{\psi}'_{\theta,x} - \sin \alpha \zeta \bar{\psi}'_\theta - n\zeta \bar{\psi}'_x) + (\eta_{13}$$

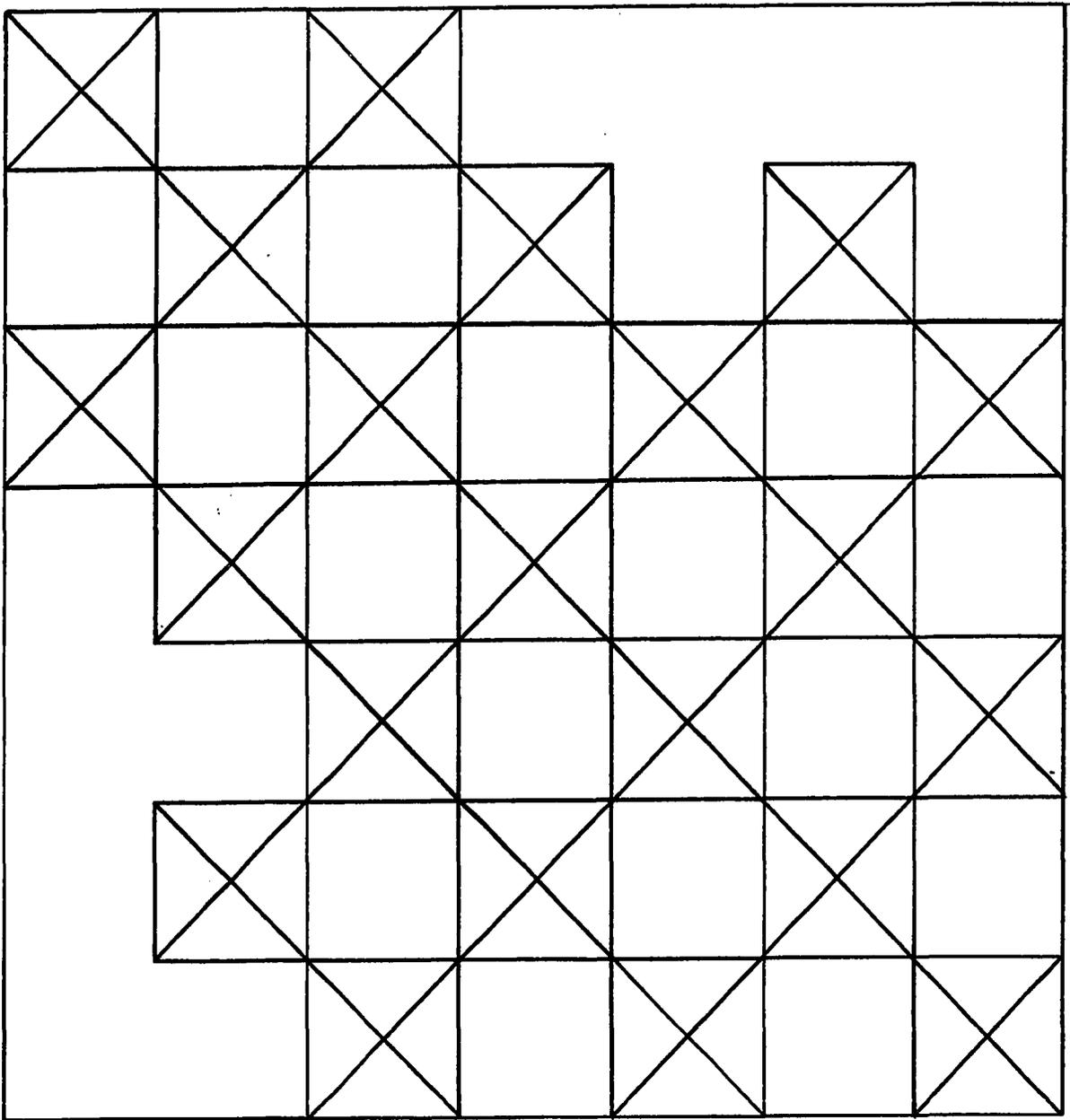


Figure 2.2  $\gamma$  Form of Stiffness Matrix when  $n = 0$

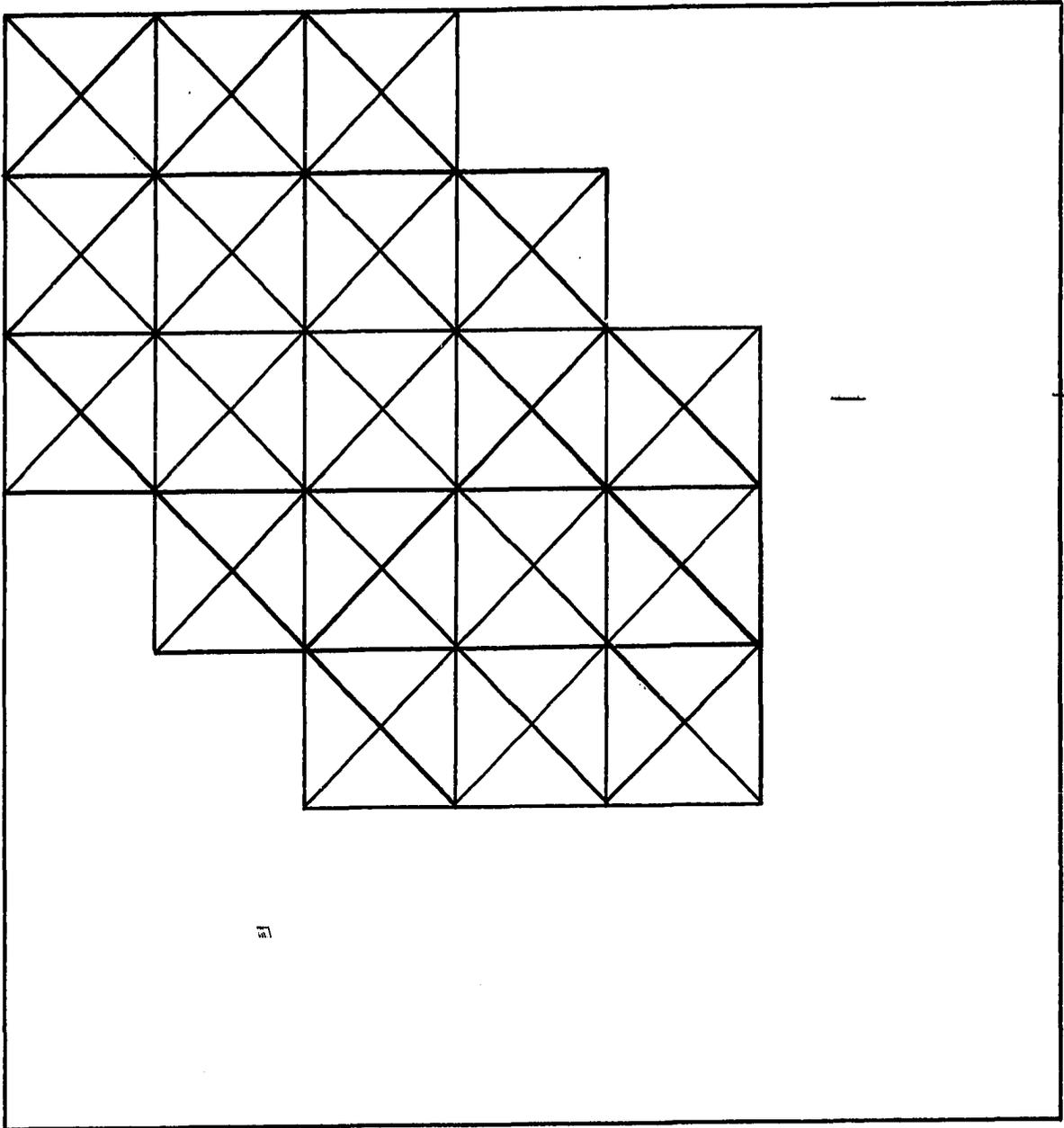


Figure 2.3 - Form of Stiffness Matrix for a Homogeneous Shell

$$+ 2h^2\eta_{12}) (\bar{\psi}_{\theta,x} - \sin \alpha \zeta \bar{\psi}_{\theta} - n\zeta \bar{\psi}_x) = 0 \quad (2-18)$$

$$\text{Either } \bar{\psi}'_x = \bar{\psi}_x = 0,$$

$$\begin{aligned} \text{or } M_x &= (2\eta_6 + \eta_9) \bar{\psi}'_{x,x} + (\eta_8 + \eta_{10}) (\sin \alpha \zeta \bar{\psi}'_x + n\zeta \bar{\psi}'_{\theta}) \\ &+ (\eta_9 + 2h^2\eta_1) \bar{\psi}_{x,x} + (\eta_{10} + h^2\eta_3) (\sin \alpha \zeta \bar{\psi}_x \\ &+ n\zeta \bar{\psi}_{\theta}) = 0 \end{aligned} \quad (2-19)$$

The freely supported boundary condition is defined here as zero displacement in the circumferential and normal directions and zero meridional stress resultant and moment at each end of the shell. Thus,

$$V = W = \bar{\psi}'_{\theta} = \bar{\psi}_{\theta} = F_x = M_x = 0. \quad (2-20)$$

The assumed solutions for  $V$ ,  $W$ ,  $\bar{\psi}_{\theta}$ , and  $\bar{\psi}'_{\theta}$  can conveniently use a series of trigonometric sines. The solutions for  $U$ ,  $\bar{\psi}'_x$ , and  $\bar{\psi}_x$  must be obtained from the conditions on  $F_x$  and  $M_x$  as follows. From the second of Equations (2-15), with  $V = W = 0$ , using the definitions of  $\eta_1$  and  $\eta_3$ , and dividing out the common factor  $4t\bar{E}'_x$ ,

$$U_{,x} + \nu'_{\theta x} \sin \alpha \zeta U = 0 \quad (2-21)$$

Using the definitions  $\epsilon = x/L$ ,  $\bar{U} = U/L$ , and  $\bar{R}_0 = R_0/L$ , Equation (2-21) may be written

$$\frac{d\bar{U}}{d\epsilon} = -\nu'_{\theta x} \frac{\sin \alpha \, d\epsilon}{\bar{R}_0 + \epsilon \sin \alpha} \quad (2-22)$$

A simple exercise will show that Equation (2-22) is identically satisfied at  $\epsilon = 0$  and  $\epsilon = 1$ , if

$$\bar{U} = \sum_m A_{1m} R_0^{-\nu'_{\theta x}} \cos m\pi\epsilon \quad (2-23)$$

From the second of Equations (2-19), with  $\bar{\psi}'_{\theta} = \bar{\psi}_{\theta} = 0$ , and using the definitions of  $\eta_1$ ,  $\eta_6$ , and  $\eta_9$ ,

$$\begin{aligned} & \bar{E}'_x \left( \frac{16}{3} t^3 + 4t^2 h \right) (\bar{\psi}'_{x,x} + \nu'_{\theta x} \sin \alpha \zeta \bar{\psi}'_x) \\ & + \bar{E}'_x (4th^2 + 4ht^2) (\bar{\psi}_{x,x} + \nu'_{\theta x} \sin \alpha \zeta \bar{\psi}_x) = 0 \end{aligned} \quad (2-24)$$

In order for Equation (2-24) to be satisfied for all values of  $\bar{E}'_x$ ,  $t$ , and  $h$ , both of the quantities in brackets must be zero independently. It is noted that, since both the terms in brackets are of the same form as Equation (2-21), the assumed solutions for  $\bar{\psi}'_x$  and  $\bar{\psi}_x$  must be of the same form as Equation (2-23).

The set of assumed modal functions for the freely supported boundary condition may now be written as

$$\begin{aligned} \bar{U} &= \sum_{m=1}^{M_1} A_{1m} R^{-\nu'_{\theta x}} \cos m\pi\epsilon \\ \bar{V} &= \sum_{m=1}^{M_2} A_{2m} \sin m\pi\epsilon \\ \bar{W} &= \sum_{m=1}^{M_3} A_{3m} \sin m\pi\epsilon \\ \bar{\psi}'_{\theta} &= \sum_{m=1}^{M_4} A_{4m} \sin m\pi\epsilon \\ \bar{\psi}'_x &= \sum_{m=1}^{M_5} A_{5m} R^{-\nu'_{\theta x}} \cos m\pi\epsilon \\ \bar{\psi}_{\theta} &= \sum_{m=1}^{M_6} A_{6m} \sin m\pi\epsilon \end{aligned}$$

$$\bar{\psi}_x = \sum_{m=1}^{M_7} A_{7m} R^{-\nu} \theta'_x \cos m\pi\epsilon \quad (2-25)$$

The clamped-clamped boundary condition may be defined as zero displacement and rotation at both ends. The assumed solutions then immediately can be written as:

$$\bar{U} = \sum_{m=1}^{M_1} A_{1m} \sin m\pi\epsilon$$

$$\bar{V} = \sum_{m=1}^{M_2} A_{2m} \sin m\pi\epsilon$$

$$\bar{W} = \sum_{m=1}^{M_3} A_{3m} \sin m\pi\epsilon$$

$$\bar{\psi}'_{\theta} = \sum_{m=1}^{M_4} A_{4m} \sin m\pi\epsilon$$

$$\bar{\psi}'_x = \sum_{m=1}^{M_5} A_{5m} \sin m\pi\epsilon$$

$$\bar{\psi}_{\theta} = \sum_{m=1}^{M_6} A_{6m} \sin m\pi\epsilon$$

$$\bar{\psi}_x = \sum_{m=1}^{M_7} A_{7m} \sin m\pi\epsilon \quad (2-26)$$

For the free-free boundary condition, the forces and moments must be zero at each end, that is,

$$F_x = F_{x\theta} = Q_x = M_{x\theta} = M_x = 0 \quad (2-27)$$

These five conditions at  $\epsilon = 0$  and  $\epsilon = 1$  are sufficient to determine the seven displacements and rotations from Equations (2-15) - (2-19). It is soon found, however, that no set of simple trigonometric series will satisfy the rather involved differential equations represented by Equation (2-27). Other types of series are not satisfactory either. A simple series of hyperbolic terms is no better than a trigonometric series, and even series of free-free beam functions\* are unsuitable.

From a physical argument, one comes to the conclusion that whatever series is used for the displacements and rotations, it must not be zero at the ends. A "free end" implies, certainly, that the displacements and rotations cannot be constrained. Thus, no trigonometric sine terms may be used, since they become zero at the ends. The simplest form which is non-zero at the ends is an appropriate series of cosine terms. The result of this argument is the use of series of cosine terms for the displacements and rotations. One should be careful, however, to start the series from zero, that is, include a term  $\cos(0)\pi\epsilon$ . This allows for the rigid body displacements and rotations which are important in the vibrations of free-free shells. Thus, for the free-free boundary conditions, the series are taken as

$$\bar{U} = \sum_{m=0}^{M_1} A_{1m} \cos m\pi\epsilon$$

---

\* Exact solutions for modal shape of simple beams, as tabulated by Young and Felgar [28].

$$\begin{aligned}\bar{V} &= \sum_{m=0}^{M_2} A_{2m} \cos m\pi\epsilon \\ \bar{W} &= \sum_{m=0}^{M_3} A_{3m} \cos m\pi\epsilon \\ \bar{\psi}'_{\theta} &= \sum_{m=0}^{M_4} A_{4m} R \cos m\pi\epsilon \\ \bar{\psi}'_x &= \sum_{m=0}^{M_5} A_{5m} \cos m\pi\epsilon \\ \bar{\psi}_{\theta} &= \sum_{m=0}^{M_6} A_{6m} R \cos m\pi\epsilon \\ \bar{\psi}_x &= \sum_{m=0}^{M_7} A_{7m} \cos m\pi\epsilon\end{aligned}\tag{2-28}$$

A point should be made here about the satisfaction of the boundary conditions for  $F_x$ ,  $M_x$ , and  $M_{x\theta}$ . It is recalled that in the assumed solutions for  $\bar{U}$ ,  $\bar{\psi}'_x$ , and  $\bar{\psi}_x$  in the freely supported case, Equations (2-25),  $F_x$  and  $M_x$  were also required to be zero. There a factor of  $R^{-\nu'_{\theta}x}$  was inserted to satisfy the boundary conditions. Here, however, because of  $F_{x\theta}$  and  $Q_x$ , the insertion of  $R^{-\nu'_{\theta}x}$  will not satisfy the boundary conditions. Since  $\nu'_{\theta}x$  is small, it is believed that the mathematical complexity resulting from its inclusion is not justified for the free-free case, in view of the other approximations.\*

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\* This fact was born out later when the factor  $R^{-\nu'_{\theta}x}$  was inserted in  $\bar{U}$ ,  $\bar{\psi}'_x$ , and  $\bar{\psi}_x$ , and resulted in less than a one percent change in the calculated frequencies and modal shapes.

Also, it is noted that the factor  $\nu_{\theta x}'$  does not appear in Equation (2-18) for  $M_{x\theta}$ , so that the factor R in the assumed modes for  $\psi_{\theta}'$  and  $\psi_{\theta}$  is included to annihilate all of the  $\psi_{\theta}'$  and  $\psi_{\theta}$  terms in  $M_{x\theta}$ .

Now that the series have been selected, the integrals which arise from Equation (2-13) may be evaluated. This operation is shown in Appendices C, D, E, and F.

## CHAPTER III

### EVALUATION OF THE THEORY

#### 3.1 Homogeneous Cylinders

The theory was first evaluated for the simplest case that can be considered, which is the case of a homogeneous, isotropic cylinder.

For the freely supported case, the experiments of Bray [29] and the analysis of Soder [30] were used for comparison. Bray tested a steel cylinder with a radius of 5.84 inches and length of 11.907 inches with 0.020-inch wall thickness. He found the lowest natural frequency to be at  $n=7$  with a value of 380 Hz. The present analysis predicts a lowest natural frequency of 380.2 Hz at  $n=7$ .

As a check case for his analysis, Soder used a steel cylinder for which Hu, Gormley, and Lindholm [31] has published experimental results. The dimensions were  $R_o = 10.0$  inches and  $L = 48.0$  inches with wall thickness = 0.03 inches. A comparison between the present analysis and Soder's analysis is given in Table 3.1. The excellent agreement between the two different analytical approaches supports Soder's contention that the experimental shell was not actually freely supported, since the experimental frequencies were somewhat higher than the analyses predict.

To evaluate the clamped-clamped boundary condition for homogeneous cylinders, the analytical results of Forsberg [32] were

n	m = 1		m = 2		m = 3		m = 4	
	Present Analysis	Ref[30]	Present Analysis	Ref[30]	Present Analysis	Ref[30]	Present Analysis	Ref [30]
2	633.9	633.5						
5	159.7	159.9	483.3	483.1	961.0	960.6		
6	167.8	168.0	370.7	370.5	724.4	724.0		
7	206.5	206.0	325.5	325.1	581.4	580.8		
12	581.6	581.1	595.5	594.9	632.9	632.3	707.1	706.4
14	792.5	791.8	802.5	801.8	825.4	824.7	868.3	867.6

Table 3.1 - Frequencies for a Freely Supported Homogeneous Cylinder (Hz)

employed. For clamped ends with axial constraint, one point was picked from two of his curves. For a radius-to-thickness ratio of 100, and a length-to-radius ratio of 5, the lowest dimensionless frequency ( $\omega/\omega_0$  in Forsberg's notation) at  $n=4$  has a value of 0.065. The present analysis was run with  $R_0 = 20.0$  inches,  $L = 100.0$  inches, and  $t = 0.05$  inches ( $t = \frac{1}{4}$  of wall thickness for a homogeneous shell). Material properties for steel were used. For  $n=4$ , the lowest natural frequency was found to be 0.0668 (non-dimensionalized). To non-dimensionalize, the frequency was multiplied by  $R_0 \sqrt{\rho' (1-\nu_{\theta x}^2)/E'_x}$ . The clamped-clamped boundary condition was checked at another point for which the radius-to-thickness ratio was 20 and the length-to-radius ratio was 2. Forsberg gave the dimensionless lowest natural frequency as 0.32, at  $n=3$ . For the present program,  $R_0 = 20.0$  inches,  $L = 40.0$  inches, and  $t = 0.25$  inches. For  $n=3$ , a frequency value of 0.325 was found.

For the free-free boundary condition, a cylinder tested by Watkins and Clary [33] was used for comparison. The steel cylinder was 42 inches long with a 14-inch radius and a wall thickness of 0.007 inches. At  $n=10$ , the lowest natural frequency was reported as 32.3 Hz. The present analysis gave a value of 34.3 Hz. However, while the second lowest frequency at  $n=10$  was reported as 32.8 Hz, the present analysis gave a value of 83.1 Hz.

### 3.2 Homogeneous Cones

The theory was next evaluated for homogeneous, isotropic conical frusta. The experimental work of Weingarten [34] was used for

comparison of both the clamped-clamped and freely supported cases, since his boundary conditions (ends potted in a low-melting-point alloy) were somewhere between those two. His steel cone had  $\alpha = 20^\circ$ ,  $R_o = 2.13$  inches,  $L = 8.0$  inches, and a wall thickness of 0.020 inches. The comparison is shown in Figure 3.1.

For the free-free boundary condition, the experiments of Hu, Gormley, and Lindholm [22] were studied analytically. Specifically, the present analysis was run using data for their steel cone for which  $\alpha = 14.2^\circ$ ,  $R_o = 2.72$  inches,  $L = 13.65$  inches and  $t = 0.0025$  inches. The results, which were not very satisfactory, are given in Table 3.2.

### 3.3 Sandwich Cone

No more complicated case than that of a homogeneous cone was found with which to compare the freely supported and clamped-clamped boundary conditions. In fact, the only experimental or analytical work found that treats a sandwich cylinder or cone is the work of Bert, et al [25], previously mentioned in Chapter I. Consequently, although their work was concerned only with the free-free boundary condition, the present analysis was run using data for their shell for both the freely supported and clamped-clamped conditions. The results were quite reasonable, as shown by comparing their experimental values with the freely supported analysis, shown in Figure 3.2 and Table 3.3, and the clamped-clamped analysis shown in Figure 3.3 and Table 3.4. The shell geometry and material properties for the shell of Bert, et al, are given by the typical input data at the end of Appendix G.

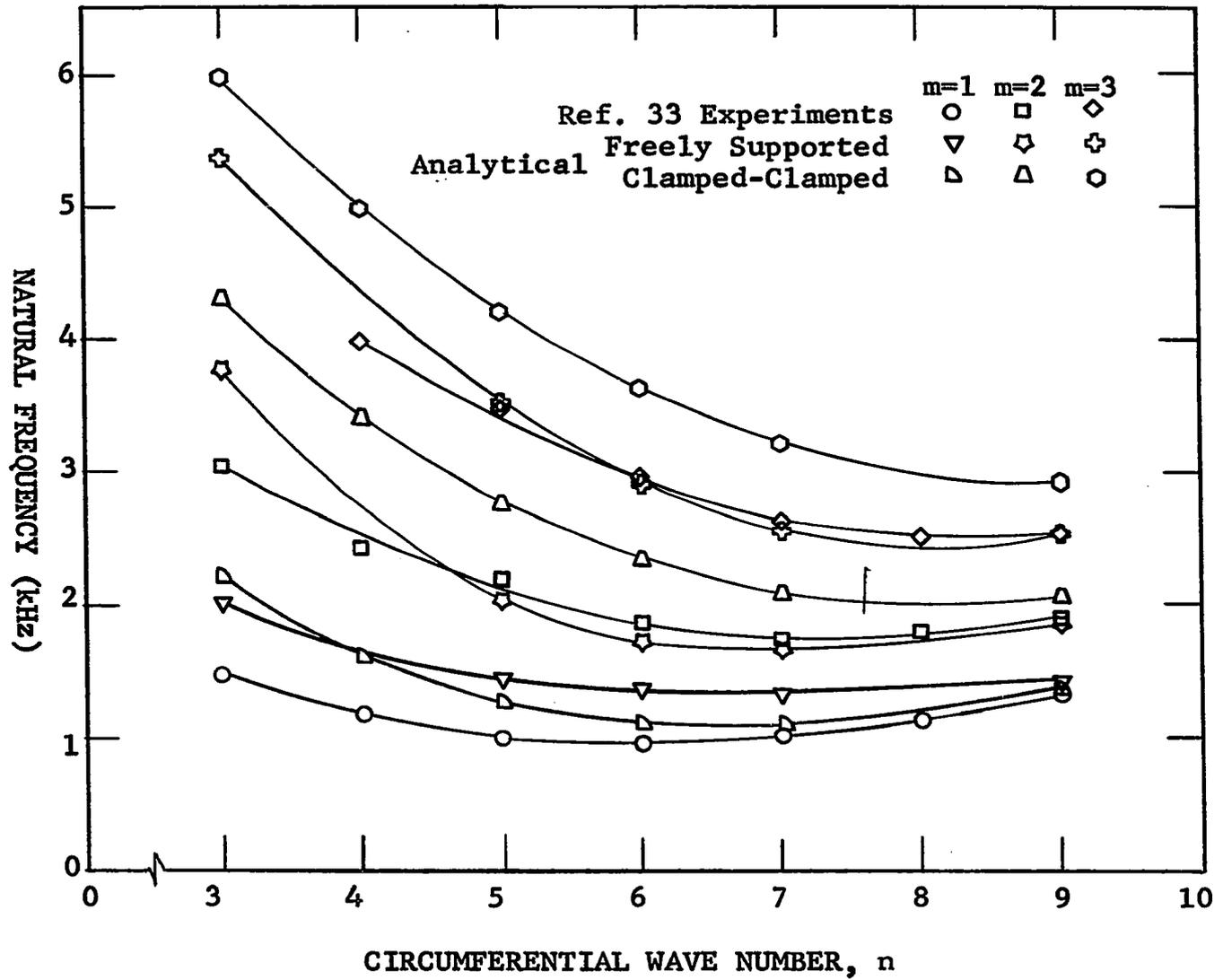


Figure 3.1 - Natural Frequencies for a Homogeneous Cone

n	m = 1		m = 2	
	Analysis	Ref. [22]	Analysis	Ref. [22]
6	349	120	974	288
8	325	200	584	385
10	391	298	634	493
12	505	423	828	622
16	1011	717	1248	959
18	1180	917		

Table 3.2 - Analytical Frequencies for Free-Free Homogeneous Cone (Hz)

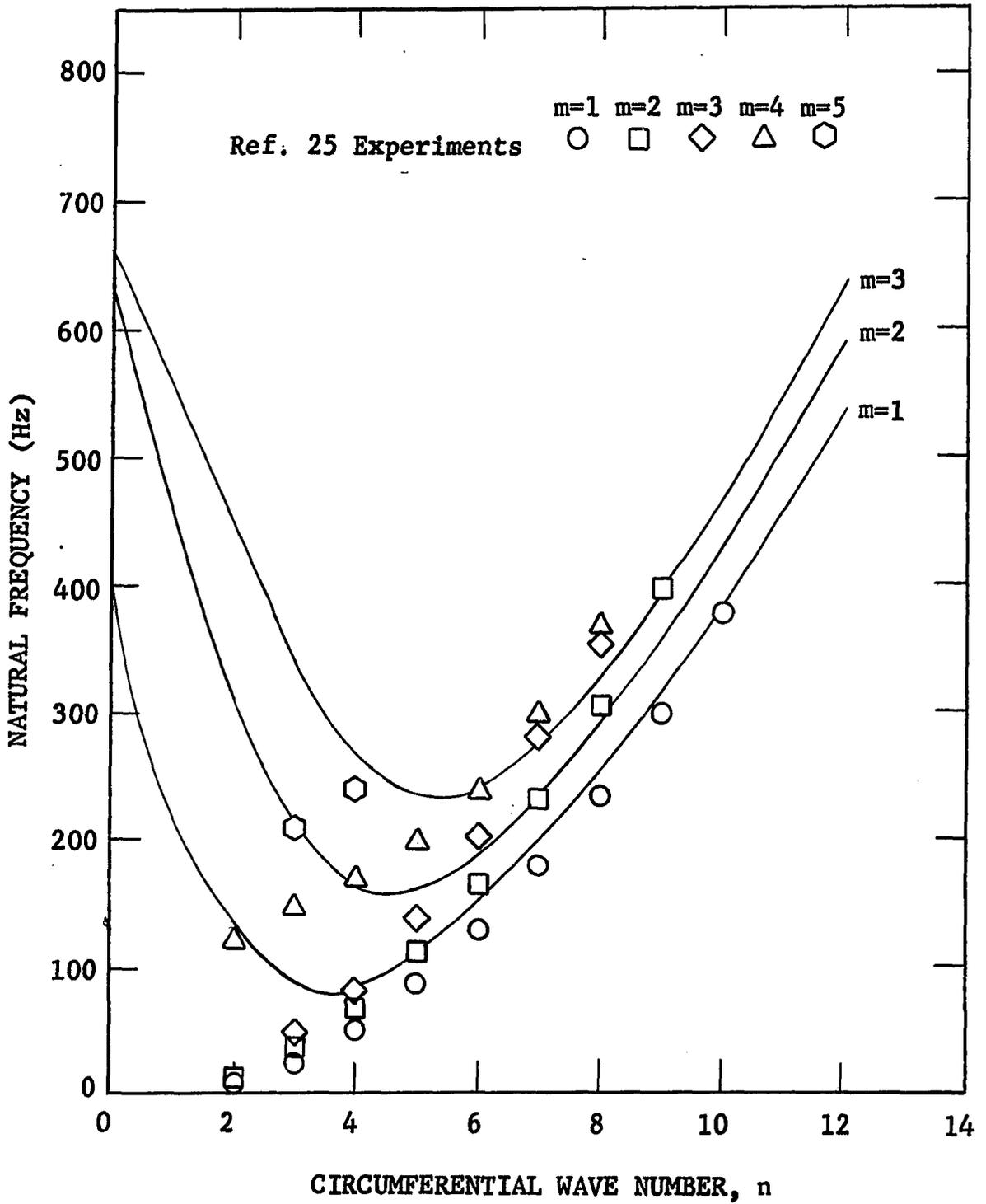


Figure 3.2 - Natural Frequencies  
for a Freely Supported Sandwich Cone

n	m = 1	m = 2	m = 3
0	406.5	624.9	655.5
2	134.8	310.9	439.9
3	86.8	212.8	331.9
4	85.7	165.4	264.7
5	113.3	160.6	235.6
6	153.7	188.9	241.5
7	201.5	236.5	275.7
8	256.2	294.1	329.5
9	317.3	358.7	395.4
10	384.6	429.7	469.3
12	535.9	589.4	636.3
20	1319.8	1412.8	1502.4

Table 3.3 - Analytical Frequencies for Freely Supported Sandwich Cone (Hz)

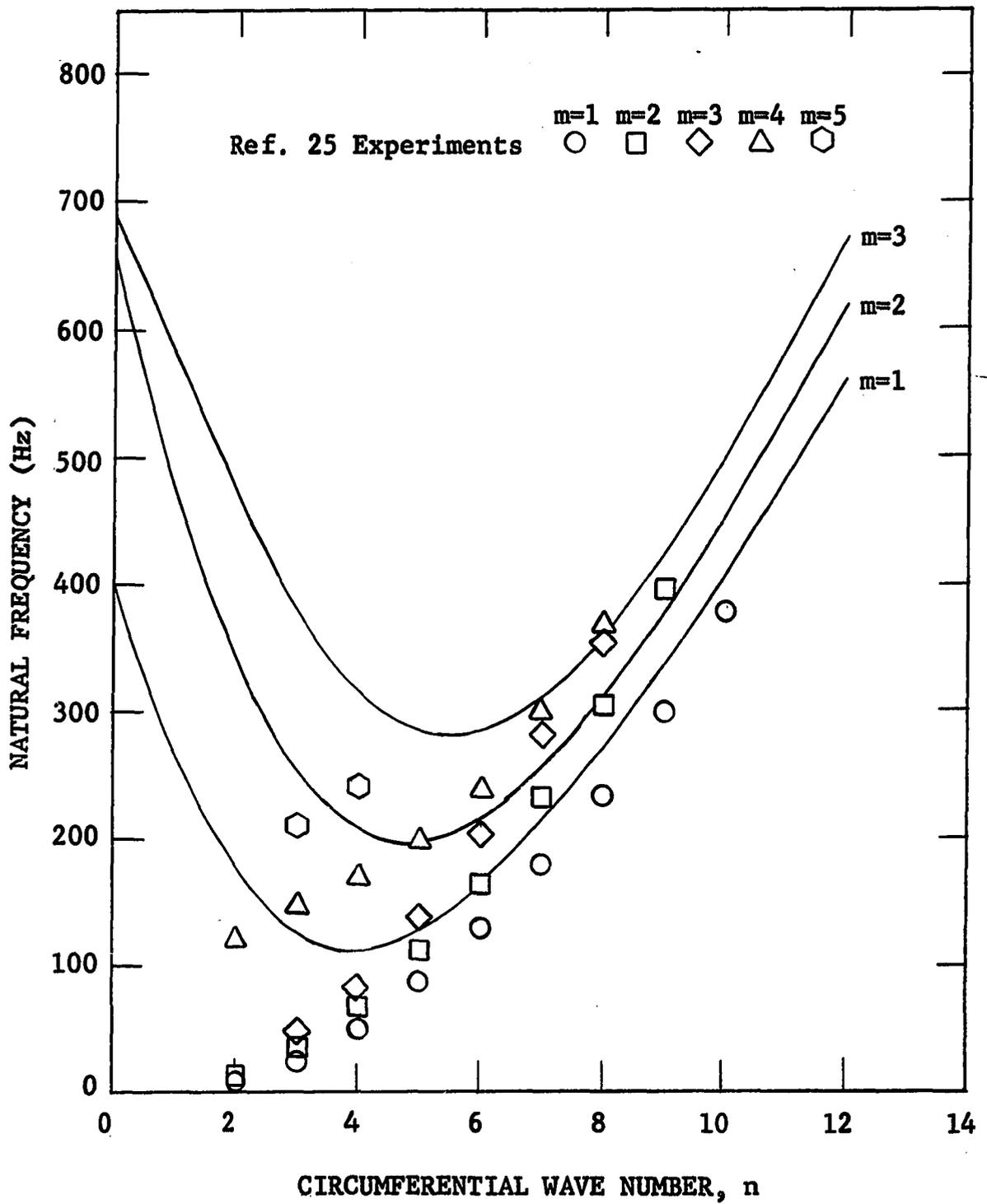


Figure 3.3 - Natural Frequencies for a Clamped-Clamped Sandwich Cone

n	m = 1	m = 2	m = 3
0	406.5	661.8	688.4
2	177.2	340.1	474.7
3	126.0	254.3	376.9
4	110.7	209.7	314.7
5	126.7	197.7	284.5
6	163.5	214.8	284.0
7	212.3	254.5	309.7
8	269.1	310.0	356.6
9	332.7	376.1	419.6
10	402.6	450.2	494.8
12	559.9	617.7	671.3
20	1368.9	1473.8	1575.9

Table 3.4 - Analytical Frequencies for Clamped-Clamped Sandwich Cone (Hz)

Of course, their data were also used to evaluate the analysis for the free-free boundary condition. The comparison and analytical values are given in Figure 3.4 and Table 3.5. The analytical modal shapes are presented in Figures 3.5 and 3.6.

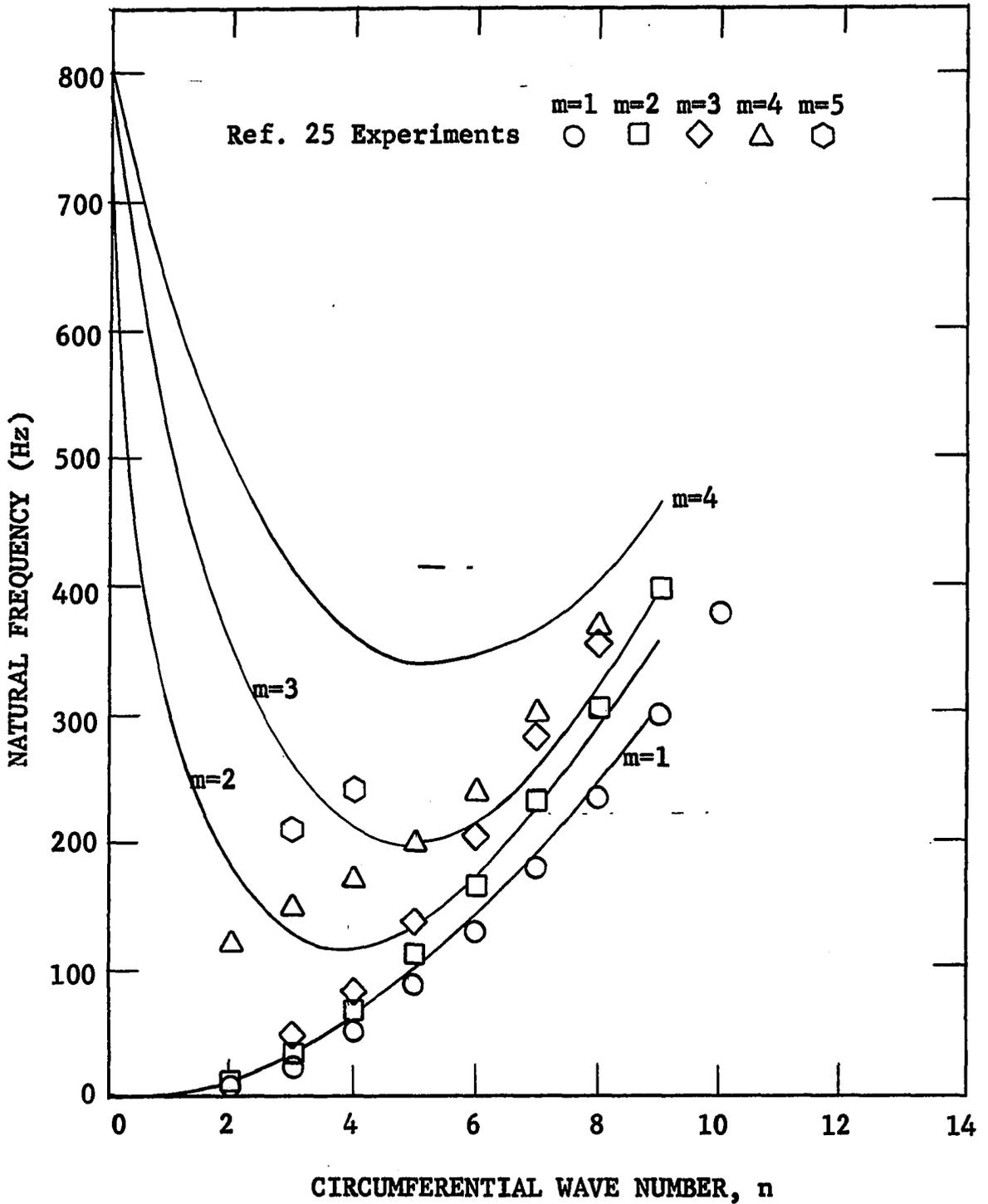


Figure 3.4 - Natural Frequencies for a Free-Free Sandwich Cone

n	m = 1	m = 2	m = 3	m = 4	m = 5	
0	0.26	724.7	790.5	805.2		Torsion modes at 32.4 and 40.8
2	13.3	179.9	384.4			
3	35.0	130.2	260.6	412.5	929.3	
4	65.3	115.7	213.1			
5	101.8	133.8	199.7	339.7	901.8	
6	143.3	175.6	217.4			
7	191.1	229.7	260.5	366.7	913.1	
8	245.7	290.5	322.5			
9	306.7	357.8	396.7	466.9	956.8	
20	1305.7	1440.7	1559.7	1689.0		

Table 3.5 - Analytical Frequencies for Free-Free Sandwich Cone (Hz)

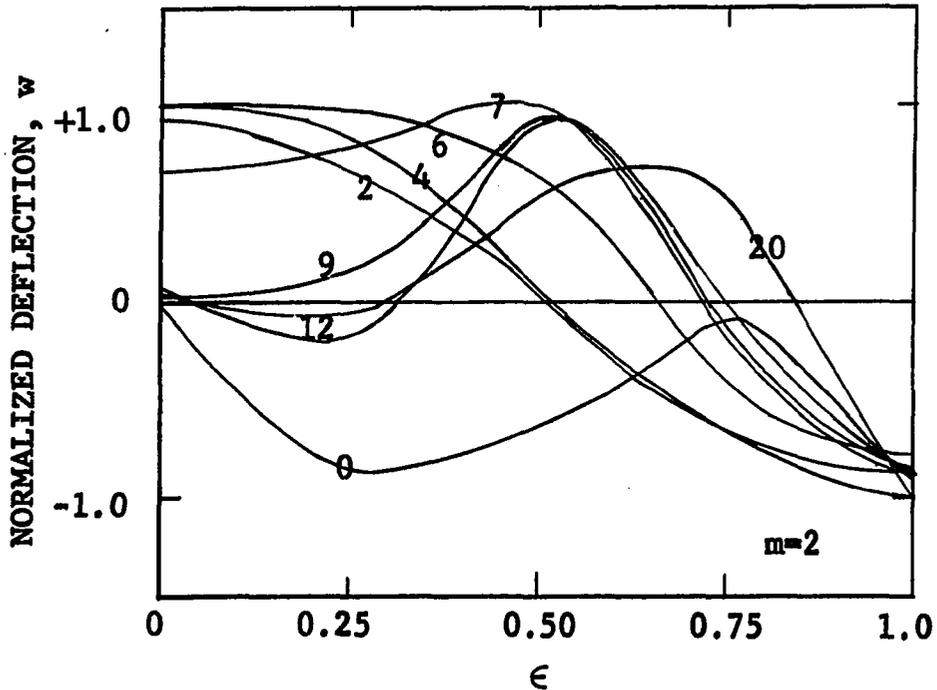
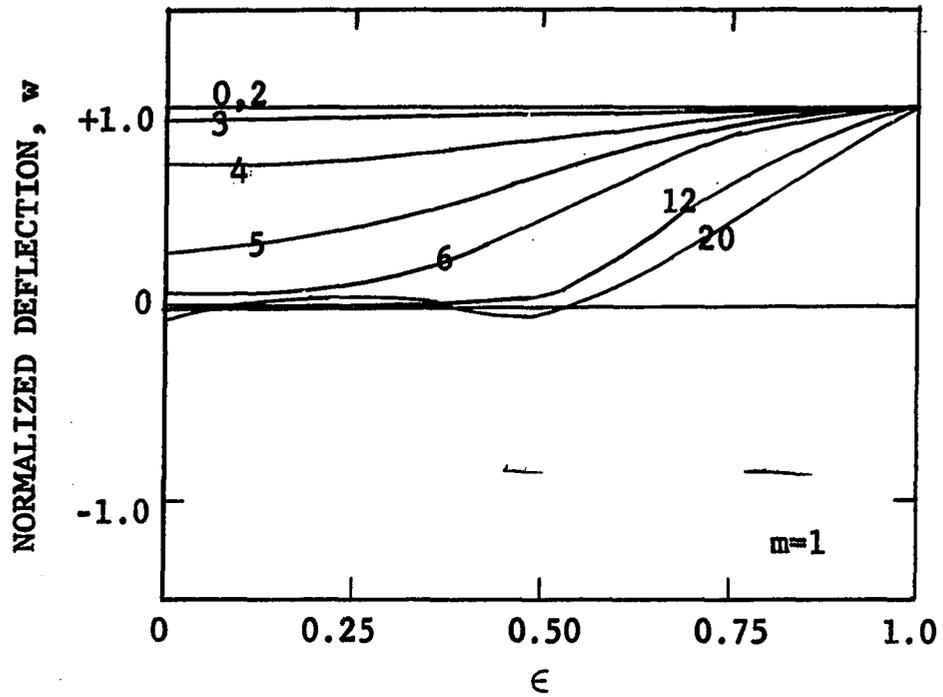


Figure 3.5 - Modal Shapes for a Free-Free Sandwich Cone with  $m=1$  and  $m=2$  and Various Values of  $n$

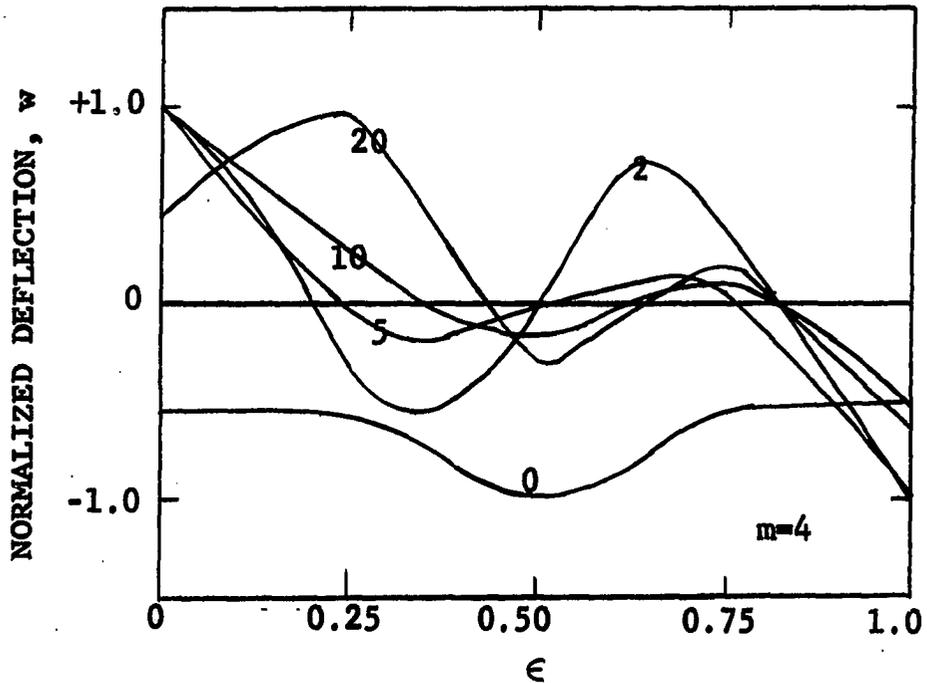
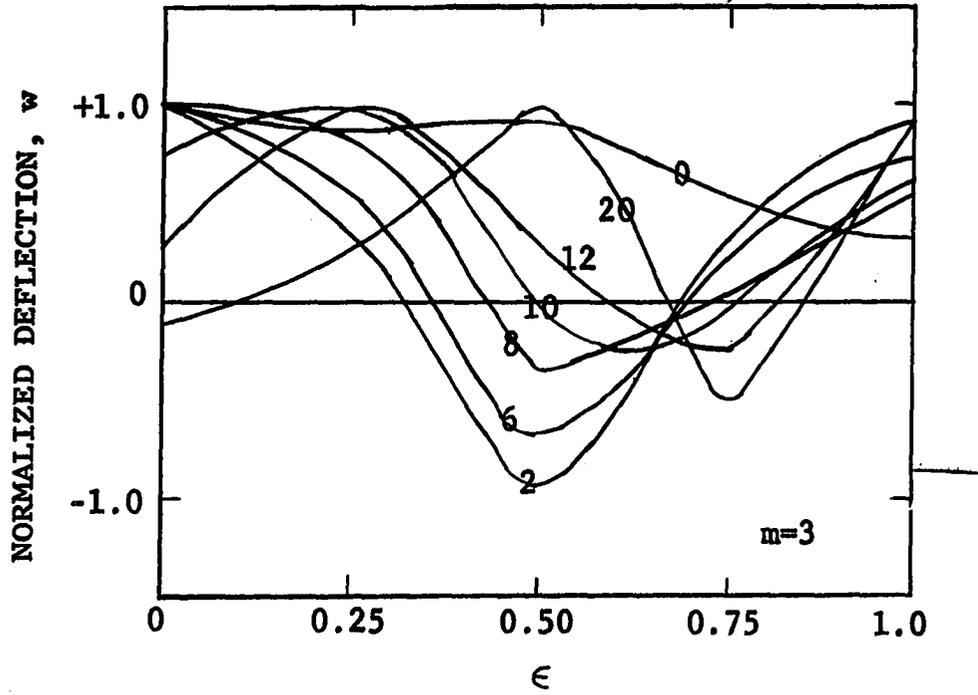


Figure 3.6 - Modal Shapes for a Free-Free Sandwich Cone with  $m=3$  and  $m=4$  and Various Values of  $n$

## CHAPTER IV

### CLOSURE

The evaluations of the present theory for the freely supported and clamped-clamped boundary conditions show generally good agreement with available experimental and analytical data.

For the free-free boundary condition, however, the theory seems to give good agreement for only the lowest natural frequency associated with each circumferential wave number, for the cases of the homogeneous cylinder and sandwich cone. It must be kept in mind that the boundary conditions were not satisfied exactly for the free-free case, so that Galerkin's method might not be expected to behave properly.

It should be noted here that there seems to be some inconsistency in the reporting of data for the free-free case. For other boundary conditions, the meridional mode number,  $m$ , can simply be thought of as indicating the number of half-waves in the deflected shape of a generator of the shell. The modes can then conveniently be identified since the lowest frequency will always have  $m=1$ , the second lowest,  $m=2$ , etc. However, upon studying the free-free modal shapes of Figures 3.5 and 3.6, it is seen that the number of half waves (and the number of nodes) varies with  $n$  for the lowest frequency, second lowest frequency, etc. This means that a simple identifying number can no longer be used, but the modal shape must be shown for each

frequency. Consequently, in this work, the designation  $m=1$  denotes only a lowest natural frequency and tells nothing about the actual modal shape. Similarly,  $m=2$  denotes the second lowest frequency, etc.

It is unfortunate that no data were available for homogeneous, orthotropic cylinders and cones, and for sandwich cylinders. When data does become available, the program is ready to handle it.

The free-free boundary condition can be pursued further when more suitable modal functions are found. Also, other boundary conditions could be rather easily investigated once modal functions are found for them.

## REFERENCES

1. Yu, Y.-Y., "Vibrations of Elastic Sandwich Cylindrical Shells", Journal of Applied Mechanics, Vol. 27, Transactions of the American Society of Mechanical Engineers, Vol. 82E, No. 4, December 1960, pp. 653-662.
2. Chu, H.N., "Vibrations of Honeycomb Sandwich Cylinders", Journal of the Aerospace Sciences, Vol. 28, No. 12, December 1961, pp. 930-939, 944.
3. Chu, H.N., "Influence of Large Amplitudes on Flexural Vibrations of a Thin Cylindrical, Sandwich Shell", Journal of the Aerospace Sciences, Vol. 29, No. 3, March 1962, p. 376.
4. Bieniek, M.P. and A.M. Freudenthal, "Forced Vibrations of Cylindrical Sandwich Shells", Journal of the Aerospace Sciences, Vol. 29, No. 2, February 1962, pp. 180-184.
5. Yu, Y.-Y., "Viscoelastic Damping of Vibrations of Sandwich Plates and Shells", Non-Classical Shell Problems (Proceedings of the International Association for Shell Structures Symposium, Warsaw, Poland, September 2-5, 1963), ed. by W. Olszak and A. Sawczuk, Amsterdam, Holland, North-Holland Publishing Company, 1964, pp. 551-571.
6. Jones, I.W. and V.L. Salerno, "The Effect of Structural Damping on the Forced Vibrations of Cylindrical Sandwich Shells", Journal of Engineering for Industry, Transactions of the American Society of Mechanical Engineers, Vol. 88B, No. 3, August 1966, pp. 318-324.
7. Greenspon, J.E., "Effect of External or Internal Static Pressure on the Natural Frequencies of Unstiffened, Cross-Stiffened, and Sandwich Cylindrical Shells", Journal of the Acoustical Society of America, Vol. 39, No. 2, February 1966, pp. 407-408.
8. Mead, D.J. and A.J. Pretlove, "On the Vibrations of Cylindrically Curved Elastic Sandwich Plates; Part 2, The Solution for Cylindrical Plates", Department of Aeronautics and Astronautics, University of Southampton; Reports and Memoranda, No. 3363, Aeronautical Research Council, London, England, 1964.

9. Jacobson, M.J. and M.L. Wenner, "Dynamic Response of Curved Composite Panels in a Thermal Environment", Norair Division, Northrop Corporation; Report 66-2647, Air Force Office of Scientific Research, Washington, D.C., November 1966, AD 642941.
10. Tasi, J., "Effect of Mass Loss on the Transient Response of a Shallow Sandwich Shell", American Institute of Aeronautics and Astronautics Journal, Vol. 2, No. 1, January 1964, pp. 58-63.
11. Koplik, B. and Y.-Y. Yu, "Axisymmetric Vibrations of Homogeneous and Sandwich Spherical Caps", Journal of Applied Mechanics, Vol. 34, Transactions of the American Society of Mechanical Engineers, Vol. 89E, No. 3, September 1967, pp. 667-673.
12. Koplik, B. and Y.-Y. Yu, "Approximate Solutions for Frequencies of Axisymmetric Vibrations of Spherical Caps", Journal of Applied Mechanics, Vol. 34, Transactions of the American Society of Mechanical Engineers, Vol. 89E, No. 3, pp. 785-787.
13. Koplik, B. and Y.-Y. Yu, "Torsional Vibrations of Homogeneous and Sandwich Caps and Circular Plates", Department of Mechanical Engineering, Polytechnic Institute of Brooklyn; Report 67-1982, Air Force Office of Scientific Research, Washington, D.C., July 1967, AD 659491.
14. Suvernev, V.G., "Vibration of Circular Conical Sandwich Shells" (in Russian), Raschety Elementiv Aviatsionnykh Konstruktsiy, Trekhsloynnye Paneli Ubutochky, Collection of Articles, No. 4, ed. by A. Ya. Aleksandrov, E.I. Grigolyuk, and L.M. Kurshin, Moscow, USSR, Izd-vo Mashinostroyeniye, 1965, pp. 91-98.
15. Klein, V., "Theory of Plates and Shells in Flight-Vehicle Design", Foreign Science Bulletin, Vol. 4, No. 2, February 1968, pp. 65-84, AD 666158.
16. Azar, J.J., "Axisymmetric Free Vibrations of Sandwich Shells of Revolution", unpublished Ph.D. dissertation, University of Oklahoma, Norman, Oklahoma, 1965.
17. Vasitsyna, T.N., "Flexure and Free Vibration of Cylindrical Sandwich Shells of Unsymmetric Construction", (in Russian) Raschety Elementov Aviatsionnykh Konstruktsiy, Trekhsloynnye Paneli i Ubutochky, Collection of Articles, No. 4, ed. by A. Ya. Aleksandrov, E.I. Grigolyuk, and L.M. Kurshin, Moscow, USSR, Izd-vo Mashinostroyeniye, 1965.
18. Baker, E.H. and G. Herrmann, "Vibrations of Orthotropic Cylindrical Sandwich Shells Under Initial Stress", American Institute of Aeronautics and Astronautics Journal, Vol. 4, No. 6, June 1966, pp. 1063-1070.

19. Bacon, M.D. and C.W. Bert, "Unsymmetric Free Vibrations of Orthotropic Sandwich Shells of Revolution", American Institute of Aeronautics and Astronautics Journal, Vol. 5, No. 3, March 1967, pp. 413-417.
20. Hu, W.C.L., "Free Vibrations of Conical Shells", Southwest Research Institute, Technical Note D-3466, National Aeronautics and Space Administration, Washington, D.C., February 1965.
21. Hu, W.C.L., J.F. Gormley, and U.S. Lindholm, "An Experimental Study and Inextensional Analysis of Vibrations of Free-Free Conical Shells", International Journal of Mechanical Sciences, Vol. 9, 1967, pp. 123-135.
22. Hu, W.C.L., J.F. Gormley, and U.S. Lindholm, Flexural Vibrations of Conical Shells with Free Edges, Contractor Report 384, National Aeronautics and Space Administration, Washington, D.C., March, 1966.
23. Mixson, J.S., "Modes of Vibration of Conical Frustum Shells with Free Ends", Journal of Spacecraft and Rockets, Vol. 4, No. 3, March 1967, pp. 414-416.
24. Krause, F.A., "Natural Frequencies and Mode Shapes of the Truncated Conical Shell with Free Edges", Technical Report 68-37, Air Force Space and Missile Systems Organization, Los Angeles Air Force Station, Los Angeles, California, January 1968, AD 665828.
25. Bert, C.W., B.L. Mayberry, and J.D. Ray, "Vibration Evaluation of Sandwich Conical Shells with Fiber-Reinforced Composite Facings", University of Oklahoma Research Institute; USAAVLABS Technical Report, U.S. Army Aviation Materiel Laboratories, Fort Eustis, Virginia, to be published.
26. Yu, Y.-Y. and J.L. Lai, "Application of Galerkin's Method to Dynamic Analysis of Structures", American Institute of Aeronautics and Astronautics Journal, Vol. 5, No. 4, 1962, p. 792.
27. Singer, J., "On the Equivalence of the Galerkin and Rayleigh-Ritz Methods", Journal of the Royal Aeronautical Society, Vol. 66, September 1962, p. 592.
28. Young, D. and R.P. Felgar, "Characteristic Functions Representing Normal Modes of Vibration of a Beam", Engineering Research Bulletin No. 4913, Bureau of Engineering Research, University of Texas, Austin, Texas, July 1949.
29. Bray, F., "Vibrations of Stiffened Cylindrical Shells", Master's thesis, University of Oklahoma, Norman, Oklahoma, in progress 1968.

30. Soder, K.E., "An Analysis of Free Vibrations of Thin Cylindrical Shells with Rings and Stringers Treated as Discrete Elements Which may be Non-Symmetric, Eccentric, and Arbitrarily Spaced", unpublished Ph.D. dissertation, University of Oklahoma, Norman, Oklahoma, 1968.
31. Hu, W.C.L., J.F. Gormley, and U.S. Lindholm, "An Analytical and Experimental Study of Vibrations of Ring-Stiffened Cylindrical Shells", Contract NASr-94(06), Technical Report No. 9, Southwest Research Institute, San Antonio, Texas, June 1967.
32. Forsberg, K., "Influence of Boundary Conditions on the Modal Characteristics of Thin Cylindrical Shells", American Institute of Aeronautics and Astronautics Journal, Vol. 2, No. 12, December 1964, pp. 2150-2157.
33. Watkins, J.D. and R.R. Clary, "Vibrational Characteristics of Some Thin-Walled Cylindrical and Conical Frustum Shells", Technical Note D-2729, National Aeronautics and Space Administration, Washington, D.C., March 1965.
34. Weingarten, V.I., "Free Vibrations of Conical Shells", Journal of the Engineering Mechanics Division, Proceedings of the American Society of Civil Engineers, Vol. 91, No. EM 4, 1965, pp. 69-87.
35. Love, A.E.H., A Treatise on the Mathematical Theory of Elasticity, Dover Publications, Inc., New York, 4th Ed., 1944.
36. System/360 Scientific Subroutine Package Version II Programmer's Manual, Form H20-0205, International Business Machines Corporation, 1968.
37. Ehrlich, L.W., "Eigenvalues and Eigenvectors of Complex Non-Hermitian Matrices Using the Direct and Inverse Power Methods and Matrix Deflation", F4 UTEX MATSUB, Co-Op User Group Distribution Agency, Control Data Corporation, Palo Alto, California, 1961.
38. System/360 Operating System Linkage Editor, Form C28-6538, International Business Machines Corporation, 1966.

## APPENDIX A

### DERIVATION OF THE KINETIC AND POTENTIAL ENERGIES FOR AN ORTHOTROPIC SANDWICH SHELL

#### A.1 Strain-Displacement Formulation

Following Love [35], the six strain components for a general orthogonal curvilinear coordinate system are

$$\begin{aligned}
 e_{\alpha\alpha} &= h_1 u_{,\alpha} + h_1 h_2 u_{\beta} (1/h_1)_{,\beta} + h_3 h_1 u_{\gamma} (1/h_1)_{,\gamma} \\
 e_{\beta\beta} &= h_2 u_{\beta,\beta} + h_2 h_3 u_{\gamma} (1/h_2)_{,\gamma} + h_1 h_2 u_{\alpha} (1/h_2)_{,\alpha} \\
 e_{\gamma\gamma} &= h_3 u_{\gamma,\gamma} + h_3 h_1 u_{\alpha} (1/h_3)_{,\alpha} + h_2 h_3 u_{\beta} (1/h_3)_{,\beta} \\
 e_{\beta\gamma} &= (h_2/h_3) (h_3 u_{\gamma})_{,\beta} + (h_3/h_2) (h_2 u_{\beta})_{,\gamma} \\
 e_{\gamma\alpha} &= (h_3/h_1) (h_1 u_{\alpha})_{,\gamma} + (h_1/h_3) (h_3 u_{\gamma})_{,\alpha} \\
 e_{\alpha\beta} &= (h_1/h_2) (h_2 u_{\beta})_{,\alpha} + (h_2/h_1) (h_1 u_{\alpha})_{,\beta}
 \end{aligned} \tag{A-1}$$

In the general coordinate system, the length of an infinitesimal line element,  $ds$ , is given by

$$(ds)^2 = (d\alpha/h_1)^2 + (d\beta/h_2)^2 + (d\gamma/h_3)^2 \tag{A-2}$$

The coordinate system for the present study is shown in Figure A.1. The middle-surface displacements are  $u$ ,  $v$ , and  $w$ , which are in the  $x$ ,  $\theta$ , and  $z$  directions, respectively.

In the present coordinate system,

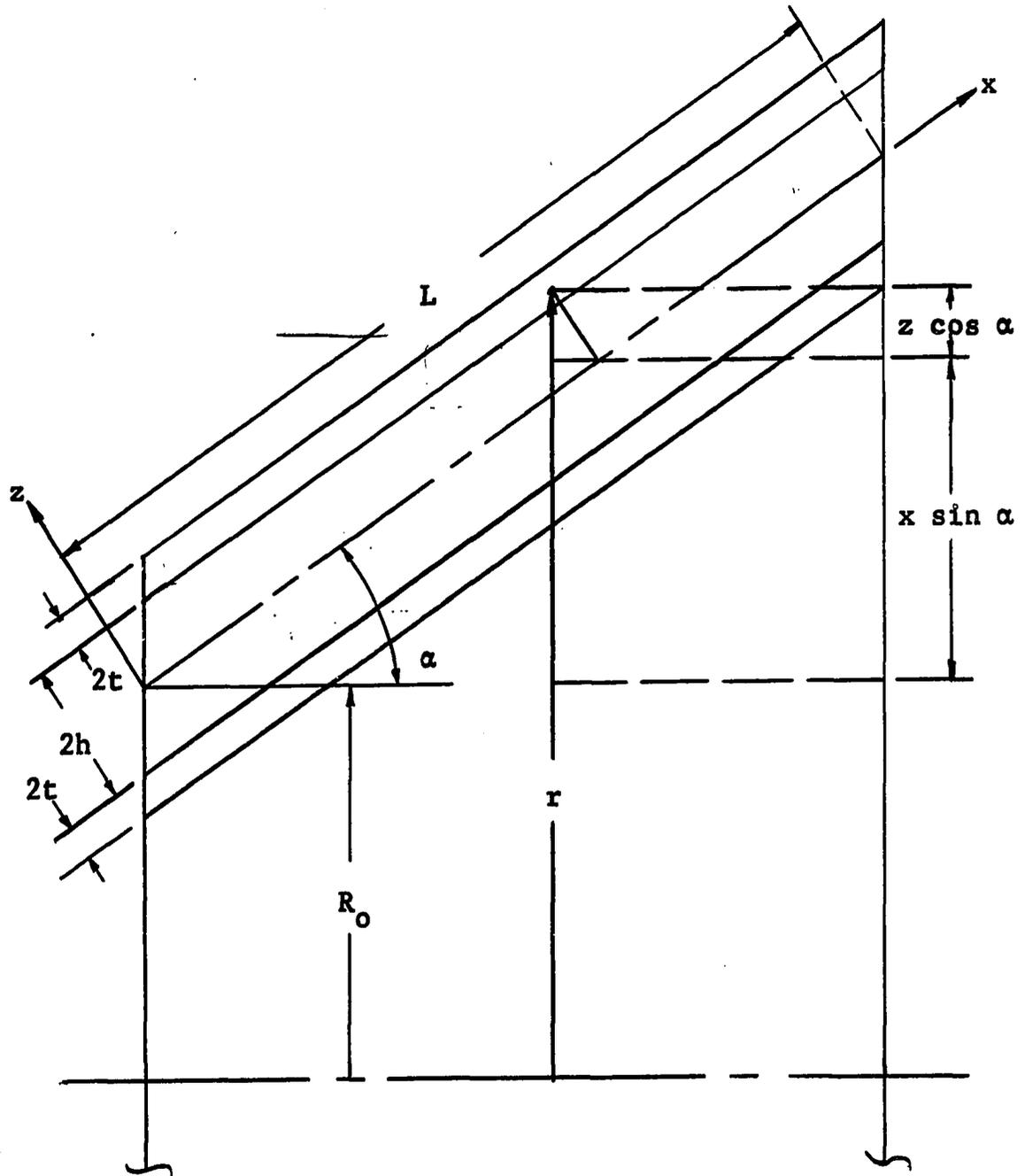


Figure A.1 - Shell Geometry

$$(ds)^2 = (dx)^2 + (rd\theta)^2 + (dz)^2 \quad (A-3)$$

Now replacing  $\alpha$  by  $x$ ,  $\beta$  by  $\theta$ , and  $\gamma$  by  $z$ , it is found that

$$(1/h_1) = 1$$

$$(1/h_2) = r \quad (A-4)$$

$$(1/h_3) = 1,$$

where  $r = R_o + x \sin \alpha + z \cos \alpha$ , and the  $\alpha$  is that shown in Figure

A.1.

Equations (A-1) now become

$$e_{xx} = u_{x,x}$$

$$e_{\theta\theta} = (1/r)(u_{\theta,\theta} + u_x \sin \alpha + u_z \cos \alpha)$$

$$e_{zz} = u_{z,z} \quad (A-5)$$

$$e_{\theta z} = (1/r)(u_{z,\theta} - u_\theta \cos \alpha) + u_{\theta,z}$$

$$e_{zx} = u_{z,x} + u_{x,z}$$

$$e_{x\theta} = (1/r)(u_{x,\theta} - u_\theta \sin \alpha) + u_{\theta,x}$$

In view of Hypothesis (6) of Section 2.2, the term  $z \cos \alpha$  in the expression for  $r$  will be neglected. In all that follows  $r$  is replaced by

$$r \sim R_o + x \sin \alpha \quad (A-6)$$

and, for later convenience, the following definition is made.

$$\zeta = (R_o + x \sin \alpha)^{-1} \quad (A-7)$$

The displacements are now defined in terms of the middle-surface displacements and the angles of rotation of normals to the middle surface in the meridional and circumferential directions. For

the core, these angles are denoted by  $\psi_x$  and  $\psi_\theta$ , while for the facings, they are  $\psi'_x$  and  $\psi'_\theta$ . The assumption is made that the core is incompressible in the thickness direction.

For the core,

$$\begin{aligned} u_x^c &= u(x, \theta, t) + z\psi_x(x, \theta, t) \\ u_\theta^c &= v(x, \theta, t) + z\psi_\theta(x, \theta, t) \\ u_z^c &= w(x, \theta, t) \end{aligned} \quad (A-8)$$

so that Equations (A-5) become, for the core,

$$\begin{aligned} e_{xx}^c &= u_{,x} + z\psi_{x,x} \\ e_{\theta\theta}^c &= \zeta(v_{,\theta} + \sin \alpha u + \cos \alpha w) \\ &\quad + z\zeta(\psi_{\theta,\theta} + \sin \alpha \psi_x) \\ e_{zz}^c &= 0 \\ e_{\theta z}^c &= \zeta(w_{,\theta} - \cos \alpha v - z \cos \alpha \psi_\theta) + \psi_\theta \\ e_{zx}^c &= w_{,x} + \psi_x \\ e_{x\theta}^c &= \zeta(u_{,\theta} - \sin \alpha v) + v_{,x} + z[\zeta(\psi_{x,\theta} - \sin \alpha \psi_\theta) \\ &\quad + \psi_{\theta,x}] \end{aligned} \quad (A-9)$$

For the outer and inner facings, respectively,

$$\begin{aligned} u_x^o, u_x^i &= u(x, \theta, t) \pm h\psi_x(x, \theta, t) + (z \mp h)\psi'_x(x, \theta, t) \\ u_\theta^o, u_\theta^i &= v(x, \theta, t) \pm h\psi_\theta(x, \theta, t) + (z \mp h)\psi'_\theta(x, \theta, t) \\ u_z^o, u_z^i &= w(x, \theta, t) \end{aligned} \quad (A-10)$$

Equations (A-5), for the facings, are now written,

$$e_{xx}^o, e_{xx}^i = u_{,x} \pm h\psi_{x,x} + (z \mp h)\psi'_{x,x}$$

$$\begin{aligned}
e_{\theta\theta}^o, e_{\theta\theta}^i &= \zeta \left\{ v_{,\theta} \pm h\psi_{\theta,\theta} + (z \mp h)\psi'_{\theta,\theta} + \sin \alpha [u \pm h\psi_{,x} \right. \\
&\quad \left. + (z \mp h)\psi'_{,x}] + \cos \alpha w \right\} \\
e_{zz}^o, e_{zz}^i &= 0 \\
e_{\theta z}^o, e_{\theta z}^i &= \zeta \left\{ w_{,\theta} - \cos \alpha [v \pm h\psi_{\theta} + (z \mp h)\psi'_{\theta}] \right\} + \psi'_{\theta} \\
e_{zx}^o, e_{zx}^i &= w_{,x} + \psi'_{,x} \\
e_{x\theta}^o, e_{x\theta}^i &= \zeta \left\{ u_{,\theta} \pm h\psi_{,x,\theta} + (z \mp h)\psi'_{,x,\theta} - \sin \alpha [v \pm h\psi_{\theta} \right. \\
&\quad \left. + (z \mp h)\psi'_{\theta}] \right\} + v_{,x} \pm h\psi_{\theta,x} + (z \mp h)\psi'_{\theta,x} \quad (A-11)
\end{aligned}$$

### A.2 Core Strain Energy

Due to Hypothesis (1), Sec. 2.2, the core strain energy is the energy due to transverse shear strain only, so that

$$V^c = \frac{1}{2} \int_x \int_{\theta} \int_z (\sigma_{zx}^c e_{zx}^c + \sigma_{\theta z}^c e_{\theta z}^c) dz \zeta^{-1} d\theta dx, \quad (A-12)$$

or

$$V^c = \frac{1}{2} \int_x \int_{\theta} \int_{-h}^h [G_{zx} (e_{zx}^c)^2 + G_{\theta z} (e_{\theta z}^c)^2] dz \zeta^{-1} d\theta dx \quad (A-13)$$

Squaring  $e_{zx}^c$  and  $e_{\theta z}^c$  from Equations (A-9), and integrating over  $z$ , gives

$$\begin{aligned}
V^c &= \int_x \int_{\theta} [hK_x G_{zx} (w_{,x}^2 + 2w_{,x}\psi_{,x} + \psi_{,x}^2) + hK_{\theta} G_{\theta z} (\zeta^2 w_{,\theta}^2 \\
&\quad + \zeta^2 \cos^2 \alpha v^2 + \{1 + (h^2 \zeta^2 \cos^2 \alpha)/3\} \psi_{\theta}^2 \\
&\quad - 2\zeta^2 \cos \alpha vw_{,\theta} + 2\zeta w_{,\theta}\psi_{\theta} - 2\zeta \cos \alpha v\psi_{\theta}] \zeta^{-1} d\theta dx \quad (A-14)
\end{aligned}$$

It is noted that the term in braces in Equation (A-14) is hereafter replaced by 1, since  $h^2 \zeta^2 \ll 1$ .

### A.3 Facing Strain Energy

Since the facings resist bending, extension, in-plane shear and transverse shear, all five non-zero strain components contribute to the strain energy. Formally written,

$$\begin{aligned}
 V^f = \frac{1}{2} \int_x \int_\theta \int_z & (\sigma_{xx}^o e_{xx}^o + \sigma_{xx}^i e_{xx}^i + \sigma_{\theta\theta}^o e_{\theta\theta}^o + \sigma_{\theta\theta}^i e_{\theta\theta}^i \\
 & + \sigma_{x\theta}^o e_{x\theta}^o + \sigma_{x\theta}^i e_{x\theta}^i + \sigma_{\theta z}^o e_{\theta z}^o + \sigma_{\theta z}^i e_{\theta z}^i + \sigma_{zx}^o e_{zx}^o \\
 & + \sigma_{zx}^i e_{zx}^i) dz \zeta^{-1} d\theta dx
 \end{aligned} \tag{A-15}$$

The necessary stress-strain relations are given by

$$\begin{aligned}
 \sigma_{xx} &= \bar{E}'_x (e_{xx} + \nu'_{\theta x} e_{\theta\theta}) & \sigma_{x\theta} &= G'_{x\theta} e_{x\theta} \\
 \sigma_{\theta\theta} &= \bar{E}'_\theta (e_{xx} + \nu'_{x\theta} e_{xx}) & \sigma_{\theta z} &= G'_{\theta x} e_{\theta z} \\
 \sigma_{zx} &= G'_{zx} e_{zx}
 \end{aligned} \tag{A-16}$$

where  $\bar{E}'_x = E'_x / (1 - \nu'_{\theta x} \nu'_{x\theta})$  and  $\bar{E}'_\theta = E'_\theta / (1 - \nu'_{\theta x} \nu'_{x\theta})$ \*. The superscripts o and i have been omitted in Equations (A-16) since they apply both to the outer and inner facings.

Substitution of Equations (A-16) into Equation (A-15) results in

$$\begin{aligned}
 V^f = \frac{1}{2} \int_x \int_\theta \int_z & \left\{ \bar{E}'_x [(e_{xx}^o)^2 + (e_{xx}^i)^2] + \bar{E}'_\theta [(e_{\theta\theta}^o)^2 + (e_{\theta\theta}^i)^2] \right. \\
 & + (\bar{E}'_x \nu'_{\theta x} + \bar{E}'_\theta \nu'_{x\theta}) (e_{xx}^o e_{\theta\theta}^o + e_{xx}^i e_{\theta\theta}^i) + G'_{x\theta} [(e_{x\theta}^o)^2 \\
 & + (e_{x\theta}^i)^2] + G'_{\theta z} [(e_{\theta z}^o)^2 + (e_{\theta z}^i)^2] + G'_{zx} [(e_{zx}^o)^2 \\
 & \left. + (e_{zx}^i)^2] \right\} dz \zeta^{-1} d\theta dx
 \end{aligned} \tag{A-17}$$

\* In an orthotropic material,  $E'_x \nu'_{\theta x} \equiv E'_\theta \nu'_{x\theta}$ .

The strains are now substituted from Equations (A-11) and the integration over  $z$  is performed, with the inner facing terms integrated from  $-h-2t$  to  $-h$ , and the outer facing terms integrated from  $h$  to  $h+2t$ .

The following integrals are necessary.

$$\int_h^{h+2t} dz = + \int_{-h-2t}^{-h} dz = + 2t \quad (\text{A-18a})$$

$$\int_h^{h+2t} z dz = - \int_{-h-2t}^{-h} z dz = + 2t(h+t) \quad (\text{A-18b})$$

$$\int_h^{h+2t} z^2 dz = + \int_{-h-2t}^{-h} z^2 dz = + \frac{2t}{3} (3h^2 + 6ht + 4t^2) \quad (\text{A-18c})$$

The expression for the facing strain energy is thus obtained

as

$$\begin{aligned} V^f = & \int_x \int_\theta \left\{ 2t\bar{E}'_x [u'_{,x}{}^2 + h^2 \psi'_{x,x}{}^2] + 4ht^2 \bar{E}'_x [\psi'_{x,x} \psi'_{x,x}] \right. \\ & + (8/3)t^3 \bar{E}'_x [\psi'_{x,x}{}^2] + 2t\bar{E}'_\theta \zeta^2 [v'_{,\theta}{}^2 + h^2 \psi'_{\theta,\theta}{}^2 + \sin^2 \alpha u'^2 \\ & + h^2 \sin^2 \alpha \psi'_{,x}{}^2 + \cos^2 \alpha w'^2 + 2 \sin \alpha v'_{,\theta} u' + 2 \cos \alpha v'_{,\theta} w' \\ & + 2h^2 \sin \alpha \psi'_{\theta,\theta} \psi'_{,x} + 2 \sin \alpha \cos \alpha u' w'] + 4ht^2 \bar{E}'_\theta \zeta^2 [\psi'_{\theta,\theta} \psi'_{\theta,\theta}] \\ & + \sin \alpha \psi'_{\theta,\theta} \psi'_{,x} + \sin \alpha \psi'_{\theta,\theta} \psi'_{,x} + \sin^2 \alpha \psi'_{,x} \psi'_{,x}] \\ & + (8/3)t^3 \bar{E}'_\theta \zeta^2 [\psi'_{\theta,\theta}{}^2 + \sin^2 \alpha \psi'_{,x}{}^2 + 2 \sin \alpha \psi'_{\theta,\theta} \psi'_{,x}] \\ & + 2t(\bar{E}'_x v'_{,\theta} + \bar{E}'_\theta v'_{,x}) \zeta [u'_{,x} v'_{,\theta} + \sin \alpha u'_{,x} u' + \cos \alpha u'_{,x} w' \\ & + h^2 \psi'_{x,x} \psi'_{\theta,\theta} + h^2 \sin \alpha \psi'_{x,x} \psi'_{,x}] + 2ht^2 (\bar{E}'_x v'_{,\theta} \\ & + \bar{E}'_\theta v'_{,x}) \zeta [\psi'_{x,x} \psi'_{\theta,\theta} + \psi'_{x,x} \psi'_{\theta,\theta} + \sin \alpha (\psi'_{x,x} \psi'_{,x} \end{aligned}$$

$$\begin{aligned}
& + \psi'_{x,x} \psi'_x] + (8/3)t^3 (\bar{E}'_{x\theta x} \nu'_{\theta x} + \bar{E}'_{\theta x\theta} \nu'_{x\theta}) \zeta [\psi'_{x,x} \psi'_{\theta,\theta} \\
& + \sin \alpha \psi'_{x,x} \psi'_x] + 2tG'_{x\theta} [\zeta^2 u_{,\theta}^2 + \zeta^2 h^2 \psi_{x,\theta}^2 + \zeta^2 \sin^2 \alpha v^2 \\
& + \zeta^2 h^2 \sin^2 \alpha \psi_{\theta}^2 + v_{,x}^2 + h^2 \psi_{\theta,x}^2 - 2\zeta^2 \sin \alpha u_{,\theta} v \\
& + 2\zeta u_{,\theta} v_{,x} - 2\zeta^2 h^2 \sin \alpha \psi_{x,\theta} \psi_{\theta} + 2\zeta h^2 \psi_{x,\theta} \psi_{\theta,x} \\
& - 2\zeta \sin \alpha v v_{,x} - 2\zeta h^2 \sin \alpha \psi_{\theta} \psi_{\theta,x}] + 4ht^2 G'_{x\theta} [\zeta^2 \psi_{x,\theta} \psi'_{x,\theta} \\
& - \zeta^2 \sin \alpha \psi_{x,\theta} \psi'_{\theta} + \zeta \psi_{x,\theta} \psi'_{\theta,x} - \zeta^2 \sin \alpha \psi'_{x,\theta} \psi_{\theta} \\
& + \zeta \psi'_{x,\theta} \psi_{\theta,x} + \zeta^2 \sin^2 \alpha \psi_{\theta} \psi'_{\theta} - \zeta \sin \alpha \psi_{\theta} \psi'_{\theta,x} \\
& - \zeta \sin \alpha \psi'_{\theta} \psi_{\theta,x} + \psi_{\theta,x} \psi'_{\theta,x}] + (8/3)t^3 G'_{x\theta} [\zeta^2 \psi_{x,\theta}^2 \\
& + \zeta^2 \sin^2 \alpha \psi_{\theta}^2 + \psi_{\theta,x}^2 - 2\zeta^2 \sin \alpha \psi'_{x,\theta} \psi'_{\theta} \\
& + 2\zeta \psi'_{x,\theta} \psi'_{\theta,x} - 2\zeta \sin \alpha \psi'_{\theta} \psi'_{\theta,x}] + 2tK'_{\theta} G'_{\theta z} [\zeta^2 w_{,\theta}^2 \\
& + \zeta^2 \cos^2 \alpha v^2 + \zeta^2 \cos^2 \alpha h^2 \psi_{\theta}^2 + \psi_{\theta}^2 - 2\zeta^2 \cos \alpha w_{,\theta} \\
& + 2\zeta \psi_{\theta} w_{,\theta} - 2\zeta \cos \alpha v \psi'_{\theta} + 2ht\zeta^2 \cos^2 \alpha \psi_{\theta} \psi'_{\theta}] \\
& + 2tK'_{xzx} G'_{zx} [w_{,x}^2 + 2w_{,x} \psi'_{x} + \psi_{x'}^2] \zeta^{-1} d\theta dx. \tag{A-19}
\end{aligned}$$

#### A.4 Total Strain Energy

The total strain energy is formed by adding Equation (A-14) and Equation (A-19), collecting like terms, and making use of the following definitions.

$$\eta_1 = 2t\bar{E}'_x$$

$$\eta_3 = 2t(\bar{E}'_{x\theta x} \nu'_{\theta x} + \bar{E}'_{\theta x\theta} \nu'_{x\theta})$$

$$\eta_2 = 2t\bar{E}'_{\theta}$$

$$\eta_4 = 2tK'_{xzx} G'_{zx}$$

$$\eta_5 = 2t\kappa'_\theta G'_{\theta z}$$

$$\eta_{11} = 2ht\eta_2$$

$$\eta_6 = \frac{4t^2}{3} \eta_1$$

$$\eta_{12} = 2tG'_{x\theta}$$

$$\eta_7 = \frac{4t^2}{3} \eta_2$$

$$\eta_{13} = 2ht\eta_{12}$$

$$\eta_8 = \frac{4t^2}{3} \eta_3$$

$$\eta_{14} = \frac{4t^2}{3} \eta_{12}$$

$$\eta_9 = 2ht\eta_1$$

$$\eta_{15} = h\kappa'_x G_{zx}$$

$$\eta_{10} = ht\eta_3$$

$$\eta_{16} = h\kappa'_\theta G_{\theta z} \quad (A-20)$$

The result is written as:

$$\begin{aligned} v = \int_x \int_\theta \{ & \eta_1 [u^2_{,x} + h^2 \psi^2_{x,x}] + \eta_2 \zeta^2 [v^2_{,\theta} + h^2 \psi^2_{\theta,\theta} + \sin^2 \alpha u^2 \\ & + h^2 \sin^2 \alpha \psi^2_x + \cos^2 \alpha w^2 + 2 \sin \alpha v_{,\theta} u \\ & + 2 \cos \alpha v_{,\theta} w + 2h^2 \sin \alpha \psi_{\theta,\theta} \psi_x + 2 \sin \alpha \cos \alpha uw] \\ & + \eta_3 \zeta [u_{,x} v_{,\theta} + \sin \alpha u_{,x} u + \cos \alpha u_{,x} w + h^2 \psi_{x,x} \psi_{\theta,\theta} \\ & + h^2 \sin \alpha \psi_{x,x} \psi_x] + \eta_4 [w^2_{,x} + 2w_{,x} \psi'_x + \psi'^2_x] + \eta_5 [\zeta^2 w^2_{,\theta} \\ & + \zeta^2 \cos^2 \alpha v^2 + \zeta^2 \cos^2 \alpha h^2 \psi^2_{\theta} + \psi'^2_{\theta} - 2\zeta^2 \cos \alpha v w_{,\theta} \\ & + 2\zeta \psi'_\theta w_{,\theta} - 2\zeta \cos \alpha v \psi'_\theta + 2ht\zeta^2 \cos^2 \alpha \psi_{\theta} \psi'_\theta] \\ & + \eta_6 [\psi'^2_{x,x}] + \eta_7 \zeta^2 [\psi'^2_{\theta,\theta} + \sin^2 \alpha \psi'^2_x + 2 \sin \alpha \psi'_{\theta,\theta} \psi'_x] \\ & + \eta_8 \zeta [\psi'_{x,x} \psi'_{\theta,\theta} + \sin \alpha \psi'_{x,x} \psi'_x] + \eta_9 [\psi'_{x,x} \psi'_{x,x}] \\ & + \eta_{10} \zeta [\psi'_{x,x} \psi'_{\theta,\theta} + \sin \alpha \psi'_{x,x} \psi'_x + \psi'_{x,x} \psi'_{\theta,\theta} \\ & + \sin \alpha \psi'_{x,x} \psi'_x] + \eta_{11} \zeta^2 [\psi'_{\theta,\theta} \psi'_{\theta,\theta} + \sin \alpha \psi'_{\theta,\theta} \psi'_x \\ & + \sin \alpha \psi'_{\theta,\theta} \psi'_x + \sin^2 \alpha \psi'_x \psi'_x] + \eta_{12} [\zeta^2 u^2_{,\theta} + \zeta^2 h^2 \psi^2_{x,\theta} + \zeta^2 \sin^2 \alpha v^2 \end{aligned}$$

$$\begin{aligned}
& + \zeta^2 h^2 \sin^2 \alpha \psi_\theta^2 + v_{,x}^2 + h^2 \psi_{\theta,x}^2 - 2\zeta^2 \sin \alpha u_{,\theta} v \\
& + 2\zeta u_{,\theta} v_{,x} - 2\zeta^2 h^2 \sin \alpha \psi_{x,\theta} \psi_\theta + 2\zeta h^2 \psi_{x,\theta} \psi_{\theta,x} \\
& - 2\zeta \sin \alpha v v_{,x} - 2\zeta h^2 \sin \alpha \psi_\theta \psi_{\theta,x} + \eta_{13} [\zeta^2 \psi_{x,\theta} \psi'_{x,\theta} \\
& - \zeta^2 \sin \alpha \psi_{x,\theta} \psi'_\theta + \zeta \psi_{x,\theta} \psi'_{\theta,x} - \zeta^2 \sin \alpha \psi'_{x,\theta} \psi_\theta \\
& + \zeta \psi'_{x,\theta} \psi_{\theta,x} + \zeta^2 \sin^2 \alpha \psi_\theta \psi'_\theta - \zeta \sin \alpha \psi_\theta \psi'_{\theta,x} \\
& - \zeta \sin \alpha \psi'_\theta \psi_{\theta,x} + \psi_{\theta,x} \psi'_{\theta,x}] + \eta_{14} [\zeta^2 \psi_{x,\theta}^2 + \zeta^2 \sin^2 \alpha \psi_\theta^2 \\
& + \psi_{\theta,x}^2 - 2\zeta^2 \sin \alpha \psi'_{x,\theta} \psi'_\theta + 2\zeta \psi'_{x,\theta} \psi'_{\theta,x} - 2\zeta \sin \alpha \psi'_\theta \psi'_{\theta,x}] \\
& + \eta_{15} [\psi_x^2 + 2\psi_x w_{,x} + w_{,x}^2] + \eta_{16} [\zeta^2 w_{,\theta}^2 + \zeta^2 \cos^2 \alpha v^2 + \psi_\theta^2 \\
& + 2\zeta w_{,\theta} \psi_\theta - 2\zeta^2 \cos \alpha w_{,\theta} v - 2\zeta \cos \alpha v \psi_\theta] \} \zeta^{-1} d\theta dx \quad (A-21)
\end{aligned}$$

#### A.5 Total Kinetic Energy

The total kinetic energy for the composite shell consists of the sum of the translational and rotatory kinetic energy of the core and facings. This can be expressed as

$$\begin{aligned}
T = \frac{1}{2} \int_x \int_\theta \{ & \bar{m}(u_{,t}^2 + v_{,t}^2 + w_{,t}^2) + J(\psi_{x,t}^2 + \psi_{\theta,t}^2) \\
& + 2J'[(\psi'_{x,t})^2 + (\psi'_{\theta,t})^2] \} \zeta^{-1} d\theta dx \quad (A-22)
\end{aligned}$$

in which

$$\begin{aligned}
\bar{m} &= 2(\rho h + 2\rho' t) \\
J &= (2/3)\rho h^3 \\
J' &= 2\rho' t[(t^2/3) + (h+t)^2] \quad (A-23)
\end{aligned}$$

APPENDIX B

APPLICATION OF HAMILTON'S PRINCIPLE TO DERIVE

THE EQUATIONS OF MOTION

Hamilton's principle requires that the first variation of the time-integrated difference between the potential and kinetic energies be zero.

$$\delta \int_{t_1}^{t_2} (V - T) dt = 0 \quad (B-1)$$

Performing this operation after combining Equation (A-21) and Equation (A-22) gives

$$\begin{aligned} \delta \int_{t_1}^{t_2} (V - T) dt = & \int_t \int_x \int_\theta \left\{ 2\eta_1 \zeta^{-1} [u_{,x} \delta u_{,x} + h^2 \psi_{x,x} \delta \psi_{x,x}] \right. \\ & + 2\eta_2 \zeta [v_{,\theta} \delta v_{,\theta} + h^2 \psi_{\theta,\theta} \delta \psi_{\theta,\theta} + \sin^2 \alpha u \delta u + h^2 \sin^2 \alpha \psi_x \delta \psi_x \\ & + \cos^2 \alpha w \delta w + \sin \alpha (v_{,\theta} \delta u + u \delta v_{,\theta}) + \cos \alpha (v_{,\theta} \delta w \\ & + w \delta v_{,\theta}) + h^2 \sin \alpha (\psi_{\theta,\theta} \delta \psi_x + \psi_x \delta \psi_{\theta,\theta}) + \sin \alpha \cos \alpha (u \delta w \\ & + w \delta u)] + \eta_3 [u_{,x} \delta v_{,\theta} + v_{,\theta} \delta u_{,x} + \sin \alpha (u_{,x} \delta u + u \delta u_{,x}) \\ & + \cos \alpha (u_{,x} \delta w + w \delta u_{,x}) + h^2 (\psi_{x,x} \delta \psi_{\theta,\theta} + \psi_{\theta,\theta} \delta \psi_{x,x}) \\ & \left. + h^2 \sin \alpha (\psi_{x,x} \delta \psi_x + \psi_x \delta \psi_{x,x}) \right] + 2\eta_4 \zeta^{-1} [w_{,x} \delta w_{,x} \end{aligned}$$

$$\begin{aligned}
& + w_{,x} \delta \psi'_x + \psi'_x \delta w_{,x} + \psi'_x \delta \psi'_x] + 2\eta_5 [\zeta v_{,\theta} \delta w_{,\theta} + \zeta \cos^2 \alpha v \delta v \\
& + \zeta h^2 \cos^2 \alpha \psi_{\theta} \delta \psi_{\theta} + \zeta^{-1} \psi'_{\theta} \delta \psi'_{\theta} - \zeta \cos \alpha (v \delta w_{,\theta} + w_{,\theta} \delta v) \\
& + (\psi'_{\theta} \delta w_{,\theta} + w_{,\theta} \delta \psi'_{\theta}) - \cos \alpha (v \delta \psi'_{\theta} + \psi'_{\theta} \delta v) \\
& + h t \zeta \cos^2 \alpha (\psi_{\theta} \delta \psi'_{\theta} + \psi'_{\theta} \delta \psi_{\theta})] + 2\eta_6 \zeta^{-1} [\psi'_{x,x} \delta \psi'_{x,x}] \\
& + 2\eta_7 \zeta [\psi'_{\theta,\theta} \delta \psi'_{\theta,\theta} + \sin^2 \alpha \psi'_{x,x} \delta \psi'_{x,x} + \sin \alpha (\psi'_{\theta,\theta} \delta \psi'_{x,x} \\
& + \psi'_{x,x} \delta \psi'_{\theta,\theta})] + \eta_8 [\psi'_{x,x} \delta \psi'_{\theta,\theta} + \psi'_{\theta,\theta} \delta \psi'_{x,x} + \sin \alpha (\psi'_{x,x} \delta \psi'_{x,x} \\
& + \psi'_{x,x} \delta \psi'_{x,x})] + \zeta^{-1} \eta_9 [\psi'_{x,x} \delta \psi_{x,x} + \psi_{x,x} \delta \psi'_{x,x}] \\
& + \eta_{10} [\psi_{x,x} \delta \psi'_{\theta,\theta} + \psi'_{\theta,\theta} \delta \psi_{x,x} + \sin \alpha (\psi_{x,x} \delta \psi'_{x,x} + \psi'_{x,x} \delta \psi_{x,x}) \\
& + \psi'_{x,x} \delta \psi_{\theta,\theta} + \psi_{\theta,\theta} \delta \psi'_{x,x} + \sin \alpha (\psi'_{x,x} \delta \psi_{x,x} + \psi_{x,x} \delta \psi'_{x,x})] \\
& + \eta_{11} \zeta [\psi_{\theta,\theta} \delta \psi'_{\theta,\theta} + \psi'_{\theta,\theta} \delta \psi_{\theta,\theta} + \sin \alpha (\psi_{\theta,\theta} \delta \psi'_{x,x} + \psi'_{x,x} \delta \psi_{\theta,\theta} \\
& + \psi'_{\theta,\theta} \delta \psi_{x,x} + \psi_{x,x} \delta \psi'_{\theta,\theta}) + \sin^2 \alpha (\psi_{x,x} \delta \psi'_{x,x} + \psi'_{x,x} \delta \psi_{x,x})] + 2\eta_{12} [\zeta u_{,\theta} \delta u_{,\theta} \\
& + \zeta h^2 \psi_{x,\theta} \delta \psi_{x,\theta} + \zeta \sin^2 \alpha v \delta v + \zeta h^2 \sin^2 \alpha \psi_{\theta} \delta \psi_{\theta} \\
& + \zeta^{-1} v_{,x} \delta v_{,x} + \zeta^{-1} h^2 \psi_{\theta,x} \delta \psi_{\theta,x} - \zeta \sin \alpha (u_{,\theta} \delta v \\
& + v \delta u_{,\theta}) + (u_{,\theta} \delta v_{,x} + v_{,x} \delta u_{,\theta}) - \zeta h^2 \sin \alpha (\psi_{x,\theta} \delta \psi_{\theta} \\
& + \psi_{\theta} \delta \psi_{x,\theta}) + h^2 (\psi_{x,\theta} \delta \psi_{\theta,x} + \psi_{\theta,x} \delta \psi_{x,\theta}) - \sin \alpha (v \delta v_{,x} \\
& + v_{,x} \delta v) - h^2 \sin \alpha (\psi_{\theta} \delta \psi_{\theta,x} + \psi_{\theta,x} \delta \psi_{\theta})] \\
& + \eta_{13} [\zeta (\psi_{x,\theta} \delta \psi'_{x,\theta} + \psi'_{x,\theta} \delta \psi_{x,\theta}) - \zeta \sin \alpha (\psi_{x,\theta} \delta \psi'_{\theta}
\end{aligned}$$

$$\begin{aligned}
& + \psi'_{\theta} \delta \psi_{x,\theta} + \psi_{x,\theta} \delta \psi'_{\theta,x} + \psi'_{\theta,x} \delta \psi_{x,\theta} - \zeta \sin \alpha (\psi'_{x,\theta} \delta \psi_{\theta} \\
& + \psi_{\theta} \delta \psi'_{x,\theta} + \psi'_{x,\theta} \delta \psi_{\theta,x} + \psi_{\theta,x} \delta \psi'_{x,\theta} + \zeta \sin^2 \alpha (\psi_{\theta} \delta \psi'_{\theta} \\
& + \psi'_{\theta} \delta \psi_{\theta}) - \sin \alpha (\psi_{\theta} \delta \psi'_{\theta,x} + \psi'_{\theta,x} \delta \psi_{\theta}) - \sin \alpha (\psi'_{\theta} \delta \psi_{\theta,x} \\
& + \psi_{\theta,x} \delta \psi'_{\theta}) + \zeta^{-1} (\psi_{\theta,x} \delta \psi'_{\theta,x} + \psi'_{\theta,x} \delta \psi_{\theta,x}) + 2\eta_{14} [\zeta \psi'_{x,\theta} \delta \psi'_{x,\theta} \\
& + \zeta \sin^2 \alpha \psi'_{\theta} \delta \psi'_{\theta} + \zeta^{-1} \psi'_{\theta,x} \delta \psi'_{\theta,x} - \zeta \sin \alpha (\psi'_{x,\theta} \delta \psi'_{\theta} \\
& + \psi'_{\theta} \delta \psi'_{x,\theta}) + \psi'_{x,\theta} \delta \psi'_{\theta,x} + \psi'_{\theta,x} \delta \psi'_{x,\theta} - \sin \alpha (\psi'_{\theta} \delta \psi'_{\theta,x} \\
& + \psi'_{\theta,x} \delta \psi'_{\theta})] + 2\eta_{15} \zeta^{-1} [\psi_x \delta \psi_x + \psi_x \delta w_x + w_x \delta \psi_x + w_x \delta w_x] \\
& + 2\eta_{16} [\zeta w_{\theta} \delta w_{\theta} + \zeta \cos^2 \alpha v \delta v + \zeta^{-1} \psi_{\theta} \delta \psi_{\theta} + w_{\theta} \delta \psi_{\theta} \\
& + \psi_{\theta} \delta w_{\theta} - \zeta \cos \alpha (w_{\theta} \delta v + v \delta w_{\theta}) - \cos \alpha (v \delta \psi_{\theta} \\
& + \psi_{\theta} \delta v)] - \bar{m} \zeta^{-1} [u_t \delta u_t + v_t \delta v_t + w_t \delta w_t] \\
& - 2J' \zeta^{-1} [\psi'_{x,t} \delta \psi'_{x,t} + \psi'_{\theta,t} \delta \psi'_{\theta,t}] - J \zeta^{-1} [\psi_{x,t} \delta \psi_{x,t} \\
& + \psi_{\theta,t} \delta \psi_{\theta,t}] \} d\theta dx dt \tag{B-2}
\end{aligned}$$

To remove terms of the form

$$\int_t \int_x f \delta(g, x) dx dt$$

they are changed to the equivalent form

$$\int_t \int_x f(\delta g),_x dx dt$$

and integrated by parts as follows

$$\int_t \int_x f(\delta g),_x dx dt = \int_t \left\{ f \delta g \Big|_{x_1}^{x_2} - \int_x f_{,x} \delta g dx \right\} dt$$

Then Equation (B-2) becomes

$$\begin{aligned}
\delta \int_{t_1}^{t_2} (V - T) dt = & \int_t \left\{ 2\eta_1 \left[ \int_{\theta} \zeta^{-1} u_{,x} \delta u d\theta \Big|_{x_1}^{x_2} - \int_x \int_{\theta} (\zeta^{-1} u_{,x})_{,x} \delta u d\theta dx \right. \right. \\
& + h^2 \int_{\theta} \zeta^{-1} \psi_{x,x} \delta \psi_x d\theta \Big|_{x_1}^{x_2} - h^2 \int_x \int_{\theta} (\zeta^{-1} \psi_{x,x})_{,x} \delta \psi_x d\theta dx \Big] \\
& + 2\eta_2 \left[ \int_x \zeta v_{,\theta} \delta v dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_{\theta} (\zeta v_{,\theta})_{,\theta} \delta v d\theta dx \right. \\
& + h^2 \int_x \zeta \psi_{\theta,\theta} \delta \psi d\theta dx \Big|_{\theta_1}^{\theta_2} - h^2 \int_x \int_{\theta} (\zeta \psi_{\theta,\theta})_{,\theta} \delta \psi d\theta dx \\
& + \sin^2 \alpha \int_x \int_{\theta} \zeta u \delta u d\theta dx + h^2 \sin^2 \alpha \int_x \int_{\theta} \zeta \psi_x \delta \psi_x d\theta dx \\
& + \cos^2 \alpha \int_x \int_{\theta} \zeta w \delta w d\theta dx + \sin \alpha \int_x \int_{\theta} \zeta v_{,\theta} \delta u d\theta dx \\
& + \sin \alpha \int_x \zeta u \delta v dx \Big|_{\theta_1}^{\theta_2} - \sin \alpha \int_x \int_{\theta} (\zeta u)_{,\theta} \delta v d\theta dx \\
& + \cos \alpha \int_x \int_{\theta} \zeta v_{,\theta} \delta w d\theta dx + \cos \alpha \int_x \zeta w \delta v dx \Big|_{\theta_1}^{\theta_2} \\
& - \cos \alpha \int_x \int_{\theta} (\zeta w)_{,\theta} \delta v d\theta dx + h^2 \sin \alpha \int_x \int_{\theta} \zeta \psi_{\theta,\theta} \delta \psi_x d\theta dx \\
& + h^2 \sin \alpha \int_x \zeta \psi_x \delta \psi_{\theta} dx \Big|_{\theta_1}^{\theta_2} - h^2 \sin \alpha \int_x \int_{\theta} (\zeta \psi_x)_{,\theta} \delta \psi_{\theta} d\theta dx \\
& + \sin \alpha \cos \alpha \int_x \int_{\theta} \zeta u \delta w d\theta dx + \sin \alpha \cos \alpha \int_x \int_{\theta} \zeta w \delta u d\theta dx \Big] \\
& + \eta_3 \left[ \int_x u_{,x} \delta v dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_{\theta} u_{,x\theta} \delta v d\theta dx + \int_{\theta} v_{,\theta} \delta u d\theta \Big|_{x_1}^{x_2} \right.
\end{aligned}$$

$$\begin{aligned}
& - \int_x \int_{\theta} v_{,x\theta} \delta u d\theta dx + \sin \alpha \int_x \int_{\theta} u_{,x} \delta u d\theta dx \\
& + \sin \alpha \int_{\theta} u \delta u d\theta \Big|_{x_1}^{x_2} - \sin \alpha \int_x \int_{\theta} u_{,x} \delta u d\theta dx \\
& + \cos \alpha \int_x \int_{\theta} u_{,x} \delta w d\theta dx + \cos \alpha \int_{\theta} w \delta u d\theta \Big|_{x_1}^{x_2} \\
& - \cos \alpha \int_x \int_{\theta} w_{,x} \delta u d\theta dx + h^2 \int_x \psi_{,x,x} \delta \psi dx \Big|_{\theta_1}^{\theta_2} \\
& - h^2 \int_x \int_{\theta} \psi_{,x\theta} \delta \psi d\theta dx + h^2 \int_{\theta} \psi_{\theta,\theta} \delta \psi dx \Big|_{x_1}^{x_2} \\
& - h^2 \int_x \int_{\theta} \psi_{\theta,x\theta} \delta \psi d\theta dx + h^2 \sin \alpha \int_{\theta} \psi_{,x} \delta \psi dx \Big|_{x_1}^{x_2} ] \\
& + 2\eta_4 \left[ \int_{\theta} \zeta^{-1} w_{,x} \delta w d\theta \Big|_{x_1}^{x_2} - \int_x \int_{\theta} (\zeta^{-1} w_{,x})_{,x} \delta w d\theta dx \right. \\
& + \int_x \int_{\theta} \zeta^{-1} w_{,x} \delta \psi'_{,x} d\theta dx + \int_{\theta} \zeta^{-1} \psi'_{,x} \delta w d\theta \Big|_{x_1}^{x_2} \\
& \left. - \int_x \int_{\theta} (\zeta^{-1} \psi'_{,x})_{,x} \delta w d\theta dx + \int_x \int_{\theta} \zeta^{-1} \psi'_{,x} \delta \psi'_{,x} d\theta dx \right] \\
& + 2\eta_5 \left[ \int_x \zeta w_{,\theta} \delta w dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_{\theta} (\zeta w_{,\theta})_{,\theta} \delta w d\theta dx \right. \\
& + \cos^2 \alpha \int_x \int_{\theta} \zeta v \delta v d\theta dx + h^2 \cos^2 \alpha \int_x \int_{\theta} \zeta \psi_{\theta} \delta \psi_{\theta} d\theta dx \\
& + \int_x \int_{\theta} \zeta^{-1} \psi'_{\theta} \delta \psi'_{\theta} d\theta dx - \cos \alpha \int_x \zeta v \delta w dx \Big|_{\theta_1}^{\theta_2} \\
& \left. + \cos \alpha \int_x \int_{\theta} (\zeta v)_{,\theta} \delta w d\theta dx - \cos \alpha \int_x \int_{\theta} \zeta w_{,\theta} \delta v d\theta dx \right]
\end{aligned}$$

$$\begin{aligned}
& + \int_x \psi'_\theta \delta w dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_\theta \psi'_{\theta, \theta} \delta w d\theta dx + \int_x \int_\theta w_{, \theta} \delta \psi'_\theta d\theta dx \\
& - \cos \alpha \int_x \int_\theta v \delta \psi'_\theta d\theta dx - \cos \alpha \int_x \int_\theta \psi'_\theta \delta v d\theta dx \\
& + ht \cos^2 \alpha \int_x \int_\theta \zeta \psi_\theta \delta \psi'_\theta d\theta dx + ht \cos^2 \alpha \int_x \int_\theta \zeta \psi'_\theta \delta \psi_\theta d\theta dx \\
& + 2\eta_6 \left[ \int_\theta \zeta^{-1} \psi'_{x, x} \delta \psi'_x d\theta \Big|_{x_1}^{x_2} - \int_x \int_\theta (\zeta^{-1} \psi'_{x, x})_{, x} \delta \psi'_x d\theta dx \right] \\
& + 2\eta_7 \left[ \int_x \zeta \psi'_{\theta, \theta} \delta \psi'_\theta dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_\theta (\zeta \psi'_{\theta, \theta})_{, \theta} \delta \psi'_\theta d\theta dx \right. \\
& + \sin^2 \alpha \int_x \int_\theta \zeta \psi'_x \delta \psi'_x d\theta dx + \sin \alpha \int_x \int_\theta \zeta \psi'_{\theta, \theta} \delta \psi'_x d\theta dx \\
& \left. + \sin \alpha \int_x \zeta \psi'_x \delta \psi'_\theta dx \Big|_{\theta_1}^{\theta_2} - \sin \alpha \int_x \int_\theta (\zeta \psi'_x)_{, \theta} \delta \psi'_\theta d\theta dx \right] \\
& + \eta_8 \left[ \int_x \psi'_{x, x} \delta \psi'_\theta dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_\theta \psi'_{x, x} \delta \psi'_\theta d\theta dx \right. \\
& + \int_\theta \psi'_{\theta, \theta} \delta \psi'_x d\theta \Big|_{x_1}^{x_2} - \int_x \int_\theta \psi'_{\theta, \theta} \delta \psi'_x d\theta dx \\
& \left. + \sin \alpha \int_\theta \psi'_x \delta \psi'_x d\theta \Big|_{x_1}^{x_2} \right] + \eta_9 \left[ \int_\theta \zeta^{-1} \psi'_{x, x} \delta \psi'_x d\theta \Big|_{x_1}^{x_2} \right. \\
& - \int_x \int_\theta (\zeta^{-1} \psi'_{x, x})_{, x} \delta \psi'_x d\theta dx + \int_\theta \zeta^{-1} \psi'_{x, x} \delta \psi'_x d\theta \Big|_{x_1}^{x_2} \\
& \left. - \int_x \int_\theta (\zeta^{-1} \psi'_{x, x})_{, x} \delta \psi'_x d\theta dx \right] + \eta_{10} \left[ \int_x \psi'_{x, x} \delta \psi'_\theta dx \Big|_{\theta_1}^{\theta_2} \right. \\
& \left. - \int_x \int_\theta \psi'_{x, x} \delta \psi'_\theta d\theta dx + \int_\theta \psi'_{\theta, \theta} \delta \psi'_x d\theta \Big|_{x_1}^{x_2} - \int_x \int_\theta \psi'_{\theta, \theta} \delta \psi'_x d\theta dx \right]
\end{aligned}$$

$$\begin{aligned}
& + \sin \alpha \int_x \int_{\theta} \psi_{x,x} \delta \psi'_x d\theta dx + \sin \alpha \int_{\theta} \psi'_x \delta \psi_x d\theta \Big|_{x_1}^{x_2} \\
& - \sin \alpha \int_x \int_{\theta} \psi'_{x,x} \delta \psi_x d\theta dx + \int_x \psi'_{x,x} \delta \psi_{\theta} dx \Big|_{\theta_1}^{\theta_2} \\
& - \int_x \int_{\theta} \psi'_{x,x\theta} \delta \psi_{\theta} d\theta dx + \int_{\theta} \psi_{\theta,\theta} \delta \psi'_x d\theta \Big|_{x_1}^{x_2} - \int_x \int_{\theta} \psi_{\theta,x\theta} \delta \psi'_x d\theta dx \\
& + \sin \alpha \int_x \int_{\theta} \psi'_{\theta,x,x} \delta \psi_x d\theta dx + \sin \alpha \int_{\theta} \psi_x \delta \psi'_x d\theta \Big|_{x_1}^{x_2} \\
& - \sin \alpha \int_x \int_{\theta} \psi_{x,x} \delta \psi'_x d\theta dx] + \eta_{11} \left[ \int_x \zeta \psi_{\theta,\theta} \delta \psi'_x dx \Big|_{\theta_1}^{\theta_2} \right. \\
& - \int_x \int_{\theta} (\zeta \psi_{\theta,\theta})_{,\theta} \delta \psi'_{\theta} d\theta dx + \int_x \zeta \psi'_{\theta,\theta} \delta \psi_{\theta} dx \Big|_{\theta_1}^{\theta_2} \\
& - \int_x \int_{\theta} \zeta \psi'_{\theta,\theta\theta} \delta \psi_{\theta} d\theta dx + \sin \alpha \int_x \int_{\theta} \zeta \psi_{\theta,\theta} \delta \psi'_x d\theta dx \\
& + \sin \alpha \int_x \zeta \psi'_x \delta \psi_{\theta} dx \Big|_{\theta_1}^{\theta_2} - \sin \alpha \int_x \int_{\theta} (\zeta \psi'_x)_{,\theta} \delta \psi_{\theta} d\theta dx \\
& + \sin \alpha \int_x \int_{\theta} \zeta \psi'_{\theta,\theta} \delta \psi'_x d\theta dx + \sin \alpha \int_x \zeta \psi_x \delta \psi'_{\theta} dx \Big|_{\theta_1}^{\theta_2} \\
& - \sin \alpha \int_x \int_{\theta} (\zeta \psi_x)_{,\theta} \delta \psi'_{\theta} d\theta dx + \sin^2 \alpha \int_x \int_{\theta} \psi_x \delta \psi'_x d\theta dx \\
& + \sin^2 \alpha \int_x \int_{\theta} \psi'_x \delta \psi_x d\theta dx] + 2\eta_{12} \left[ \int_x \zeta u_{,\theta} \delta u dx \Big|_{\theta_1}^{\theta_2} \right. \\
& - \int_x \int_{\theta} (\zeta u_{,\theta})_{,\theta} \delta u dx d\theta + h^2 \int_x \zeta \psi_{x,\theta} \delta \psi_x dx \Big|_{\theta_1}^{\theta_2} \\
& - h^2 \int_x \int_{\theta} (\zeta \psi_{x,\theta})_{,\theta} \delta \psi'_x d\theta dx + \sin^2 \alpha \int_x \int_{\theta} \zeta v \delta v d\theta dx
\end{aligned}$$

$$\begin{aligned}
& \int_0^x \int_0^x \alpha \sin \theta \psi_{\theta, \theta}^x dx + \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - \int_0^x \int_0^x \alpha \sin \theta \psi_{\theta, \theta}^x dx - \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - \int_0^x \int_0^x \psi_{\theta, \theta}^x dx + \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - h^2 \sin \alpha \int_0^x \psi_{\theta, \theta}^x dx + h^2 \int_0^x \psi_{\theta, \theta}^x dx \\
& - h^2 \int_0^x \int_0^x \alpha \sin \theta \psi_{\theta, \theta}^x dx + h^2 \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - h^2 \sin \alpha \int_0^x \int_0^x \psi_{\theta, \theta}^x dx - h^2 \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - \int_0^x \int_0^x \psi_{\theta, \theta}^x dx + \int_0^x \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - \sin \alpha \int_0^x \int_0^x \psi_{\theta, \theta}^x dx + \int_0^x \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x \\
& - h^2 \int_0^x \int_0^x (\psi_{\theta, \theta}^x)_{\theta, \theta}^x dx - \int_0^x \int_0^x \psi_{\theta, \theta}^x dx \\
& - \int_0^x \int_0^x (\psi_{\theta, \theta}^x)_{\theta, \theta}^x dx + h^2 \int_0^x \int_0^x \psi_{\theta, \theta}^x dx \\
& + h^2 \sin^2 \alpha \int_0^x \int_0^x \psi_{\theta, \theta}^x dx + \int_0^x \int_0^x \psi_{\theta, \theta}^x dx \Big|_0^x
\end{aligned}$$

$$\begin{aligned}
& - \int_x \int_\theta \psi_{x,\theta x} \delta \psi'_\theta d\theta dx + \int_x \psi'_{\theta,x} \delta \psi_x dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_\theta \psi'_{\theta,x\theta} \delta \psi_x d\theta dx \\
& - \sin \alpha \int_x \int_\theta \delta \psi'_{x,\theta} \delta \psi_\theta d\theta dx - \sin \alpha \int_x \zeta \psi_\theta \delta \psi'_x dx \Big|_{\theta_1}^{\theta_2} \\
& + \sin \alpha \int_x \int_\theta \zeta \psi_{\theta,\theta} \delta \psi'_x d\theta dx + \int_\theta \psi'_{x,\theta} \delta \psi_\theta d\theta \Big|_{x_1}^{x_2} \\
& - \int_x \int_\theta \psi'_{x,\theta} \delta \psi_\theta d\theta dx + \int_x \psi_{\theta,x} \delta \psi'_x dx \Big|_{\theta_1}^{\theta_2} - \int_x \int_\theta \psi_{\theta,x\theta} \delta \psi'_x d\theta dx \\
& + \sin^2 \alpha \int_x \int_\theta \zeta \psi_\theta \delta \psi'_\theta d\theta dx + \sin^2 \alpha \int_x \int_\theta \zeta \psi'_\theta \delta \psi_\theta d\theta dx \\
& - \sin \alpha \int_\theta \psi_\theta \delta \psi'_\theta d\theta \Big|_{x_1}^{x_2} + \sin \alpha \int_x \int_\theta \psi_{\theta,x} \delta \psi'_\theta d\theta dx - \sin \alpha \int_\theta \psi'_\theta \delta \psi_\theta d\theta \Big|_{x_1}^{x_2} \\
& - \sin \alpha \int_x \int_\theta \psi'_{\theta,x} \delta \psi_\theta d\theta dx - \sin \alpha \int_x \int_\theta \psi'_{\theta,x} \delta \psi_\theta d\theta dx \\
& - \sin \alpha \int_x \int_\theta \psi_{\theta,x} \delta \psi'_\theta d\theta dx + \int_\theta \zeta^{-1} \psi_{\theta,x} \delta \psi'_\theta d\theta \Big|_{x_1}^{x_2} \\
& - \int_x \int_\theta (\zeta^{-1} \psi_{\theta,x})_{,x} \delta \psi'_\theta d\theta dx + \int_\theta \zeta^{-1} \psi'_{\theta,x} \delta \psi_\theta d\theta \Big|_{x_1}^{x_2} \\
& - \int_x \int_\theta (\zeta^{-1} \psi'_{\theta,x})_{,x} \delta \psi_\theta d\theta dx] + 2\eta_{14} \left[ \int_x \zeta \psi'_{x,\theta} \delta \psi'_x dx \Big|_{\theta_1}^{\theta_2} \right. \\
& \left. - \int_x \int_\theta \zeta \psi'_{x,\theta\theta} \delta \psi'_x d\theta dx + \sin^2 \alpha \int_x \int_\theta \zeta \psi'_\theta \delta \psi'_\theta d\theta dx \right. \\
& \left. + \int_\theta \zeta^{-1} \psi'_{\theta,x} \delta \psi'_\theta d\theta \Big|_{x_1}^{x_2} - \int_x \int_\theta (\zeta^{-1} \psi'_{\theta,x})_{,x} \delta \psi'_\theta d\theta dx \right. \\
& \left. - \sin \alpha \int_x \int_\theta \zeta \psi'_{x,\theta} \delta \psi'_\theta d\theta dx - \sin \alpha \int_x \zeta \psi'_\theta \delta \psi'_x dx \Big|_{\theta_1}^{\theta_2} \right.
\end{aligned}$$

$$\begin{aligned}
& + \sin \alpha \int_x \int_{\theta} \zeta \psi'_{\theta, \theta} \delta \psi'_{x'} d\theta dx + \int_{\theta} \psi'_{x, \theta} \delta \psi'_{\theta} d\theta \Big|_{x_1}^{x_2} \\
& - \int_x \int_{\theta} \psi'_{x, x\theta} \delta \psi'_{\theta} d\theta dx + \int_x \psi'_{\theta, x} \delta \psi'_{x'} dx \Big|_{\theta_1}^{\theta} - \int_x \int_{\theta} \psi'_{\theta, x\theta} \delta \psi'_{x'} d\theta dx \\
& - \sin \alpha \int_{\theta} \psi'_{\theta} \delta \psi'_{\theta} d\theta \Big|_{x_1}^{x_2} + 2\eta_{15} \left[ \int_x \int_{\theta} \zeta^{-1} \psi'_{x'} \delta \psi'_{x'} d\theta dx \right. \\
& + \int_{\theta} \zeta^{-1} \psi'_{x'} \delta w d\theta \Big|_{x_1}^{x_2} - \int_x \int_{\theta} (\zeta^{-1} \psi'_{x'})_{, x} \delta w d\theta dx + \int_{\theta} \zeta^{-1} w_{, x} \delta w d\theta \Big|_{x_1}^{x_2} \\
& \left. - \int_x \int_{\theta} (\zeta^{-1} w_{, x})_{, x} \delta w d\theta dx + \int_x \int_{\theta} \zeta^{-1} w_{, x} \delta \psi'_{x'} d\theta dx \right] \\
& + 2\eta_{16} \left[ \int_x \zeta w_{, \theta} \delta w dx \Big|_{\theta_1}^{\theta} - \int_x \int_{\theta} \zeta w_{, \theta\theta} \delta w d\theta dx + \cos^2 \alpha \int_x \int_{\theta} \zeta v \delta v d\theta dx \right. \\
& + \int_x \int_{\theta} \zeta^{-1} \psi'_{\theta} \delta \psi'_{\theta} d\theta dx + \int_x \int_{\theta} w_{, \theta} \delta \psi'_{\theta} d\theta dx + \int_x \psi'_{\theta} \delta w dx \Big|_{\theta_1}^{\theta} \\
& \left. - \int_x \int_{\theta} \psi'_{\theta, \theta} \delta w d\theta dx - \cos \alpha \int_x \int_{\theta} \zeta w_{, \theta} \delta v d\theta dx - \cos \alpha \int_x \zeta v \delta w dx \Big|_{\theta_1}^{\theta} \right. \\
& + \cos \alpha \int_x \int_{\theta} \zeta v_{, \theta} \delta w d\theta dx - \cos \alpha \int_x \int_{\theta} v \delta \psi'_{\theta} d\theta dx \\
& \left. - \cos \alpha \int_x \int_{\theta} \psi'_{\theta} \delta v d\theta dx \right] dt - \bar{m} \int_x \int_{\theta} \zeta^{-1} [u_{, t} \delta u] \Big|_{t_1}^{t_2} \\
& - \int_t u_{, tt} \delta u dt + v_{, t} \delta v \Big|_{t_1}^{t_2} - \int_t v_{, tt} \delta v dt + w_{, t} \delta w \Big|_{t_1}^{t_2} \\
& - \int_t w_{, tt} \delta w dt] d\theta dx - 2J' \int_x \int_{\theta} \zeta^{-1} [\psi'_{x, t} \delta \psi'_{x'}] \Big|_{t_1}^{t_2} - \int_t \psi'_{x, tt} \delta \psi'_{x'} dt \\
& + \psi'_{\theta, t} \delta \psi'_{\theta} \Big|_{t_1}^{t_2} - \int_t \psi'_{\theta, tt} \delta \psi'_{\theta} dt] d\theta dx - J \int_x \int_{\theta} \zeta^{-1} [\psi'_{\theta, t} \delta \psi'_{\theta}] \Big|_{t_1}^{t_2}
\end{aligned}$$

$$- \int_t \psi_{\theta, tt} \delta \psi_{\theta} dt + \psi_{x, t} \delta \psi_x \Big|_{t_1}^{t_2} - \int_t \psi_{x, tt} \delta \psi_x dt] d\theta dx. \quad (B-3)$$

It is noted that since the variation of the function with  $\theta$  is periodic, all of the terms in Equation (B-3) that are shown to be evaluated at  $\theta_1$  and  $\theta_2$  reduce to zero. Also, by the definition of the method, all those terms evaluated at  $t_1$  and  $t_2$  reduce to zero.

After grouping terms and removing a factor of  $\zeta^{-1}$  which appears as part of the integration variables,

$$\begin{aligned} \delta \int_t (V-T) dt = & \int_t \int_x \int_{\theta} \{ [-2\eta_1 (u_{,xx} + \sin \alpha \zeta u_{,x}) + 2\eta_2 \sin^2 \alpha \zeta^2 u \\ & + 2\eta_2 \sin \alpha \zeta^2 v_{,\theta} + 2\eta_2 \sin \alpha \cos \alpha \zeta^2 w - \eta_3 v_{,x\theta} - \eta_3 \cos \alpha \zeta w_{,x} \\ & - 2\eta_{12} \zeta^2 u_{,\theta\theta} + 2\eta_{12} \sin \alpha \zeta^2 v_{,\theta} - 2\eta_{12} \zeta v_{,x\theta} + \bar{m} u_{,tt}] \delta u \\ & + [-2\eta_2 \zeta^2 v_{,\theta\theta} - 2\eta_2 \sin \alpha \zeta^2 u_{,\theta} - 2\eta_2 \cos \alpha \zeta^2 w_{,\theta} \\ & - \eta_3 \zeta u_{,x\theta} + 2\eta_5 \cos^2 \alpha \zeta^2 v - 2\eta_5 \cos \alpha \zeta^2 w_{,\theta} - 2\eta_5 \cos \alpha \zeta \psi'_{\theta} \\ & + 2\eta_{12} \sin^2 \alpha \zeta^2 v - 2\eta_{12} (v_{,xx} + \sin \alpha \zeta v_{,x}) - 2\eta_{12} \sin \alpha \zeta^2 u_{,\theta} \\ & - 2\eta_{12} \zeta u_{,x\theta} + 2\eta_{16} \cos^2 \alpha \zeta^2 v - 2\eta_{16} \cos \alpha \zeta^2 w_{,\theta} \\ & - 2\eta_{16} \cos \alpha \zeta \psi_{\theta} + \bar{m} v_{,tt}] \delta v + [+ 2\eta_2 \cos^2 \alpha \zeta^2 w \\ & + 2\eta_2 \cos \alpha \zeta^2 v_{,\theta} + 2\eta_2 \sin \alpha \cos \alpha \zeta^2 u + \eta_3 \cos \alpha \zeta u_{,x} \\ & - 2\eta_4 (w_{,xx} + \sin \alpha \zeta w_{,x}) - 2\eta_4 (\psi'_{x,x} \\ & + \sin \alpha \zeta \psi'_{x'}) - 2\eta_5 \zeta^2 w_{,\theta\theta} + 2\eta_5 \cos \alpha \zeta^2 v_{,\theta} - 2\eta_5 \zeta \psi'_{\theta,\theta} \\ & - 2\eta_{15} (\psi_{x,x} + \sin \alpha \zeta \psi_x) - 2\eta_{15} (w_{,xx} + \sin \alpha \zeta w_{,x}) \end{aligned}$$

$$\begin{aligned}
& - 2\eta_{16} \zeta^2 w_{,\theta\theta} - 2\eta_{16} \zeta \psi_{\theta,\theta} + 2\eta_{16} \cos \alpha \zeta^2 v_{,\theta} + \bar{m} w_{,tt} \delta w \\
& + [ + 2\eta_5 \psi'_{\theta} + 2\eta_5 \zeta w_{,\theta} - 2\eta_5 \cos \alpha \zeta v + 2\eta_5 h t \cos^2 \alpha \zeta^2 \psi_{\theta} \\
& - 2\eta_7 \zeta^2 \psi'_{\theta,\theta\theta} - 2\eta_7 \sin \alpha \zeta^2 \psi'_{x,\theta} - \eta_8 \zeta \psi'_{x,x\theta} - \eta_{10} \zeta \psi_{x,x\theta} \\
& - \eta_{11} \zeta^2 \psi_{\theta,\theta\theta} - \sin \alpha \eta_{11} \zeta^2 \psi_{x,\theta} - \eta_{13} \sin \alpha \zeta^2 \psi_{x,\theta} \\
& - \eta_{13} \zeta \psi_{x,x\theta} + \eta_{13} \sin^2 \alpha \zeta^2 \psi_{\theta} - \eta_{13} (\psi_{\theta,xx} + \sin \alpha \zeta \psi_{\theta,x}) \\
& + 2\eta_{14} \sin^2 \alpha \zeta^2 \psi'_{\theta} - 2\eta_{14} (\psi'_{\theta,xx} + \sin \alpha \zeta \psi'_{\theta,x}) \\
& - 2\eta_{14} \sin \alpha \zeta^2 \psi'_{x,\theta} - 2\eta_{14} \zeta \psi'_{x,\theta x} + 2J' \psi'_{\theta,tt} \delta \psi'_{\theta} + [ + 2\eta_4 w_{,x} \\
& + 2\eta_4 \psi'_{x} - 2\eta_6 (\psi'_{x,xx} + \sin \alpha \zeta \psi'_{x,x}) + 2\eta_7 \sin^2 \alpha \zeta^2 \psi'_{x} \\
& + 2\eta_7 \sin \alpha \zeta^2 \psi'_{\theta,\theta} - \eta_8 \zeta \psi'_{\theta,x\theta} - \eta_9 (\psi_{x,xx} + \sin \alpha \zeta \psi_{x,x}) \\
& - \eta_{10} \psi_{\theta,x\theta} + \eta_{11} \sin \alpha \zeta^2 \psi_{\theta,\theta} + \eta_{11} \sin^2 \alpha \zeta \psi_x - \eta_{13} \zeta^2 \psi_{x,\theta\theta} \\
& + \eta_{13} \sin \alpha \zeta^2 \psi_{\theta,\theta} - \eta_{13} \zeta \psi_{\theta,x\theta} - 2\eta_{14} \zeta^2 \psi'_{x,\theta\theta} \\
& + 2\eta_{14} \sin \alpha \zeta^2 \psi'_{\theta,\theta} - 2\eta_{14} \zeta \psi'_{\theta,x\theta} + 2J' \psi'_{x,tt} \delta \psi'_x \\
& + [ - 2\eta_2 h^2 \zeta^2 \psi_{\theta,\theta\theta} - 2\eta_2 h^2 \sin \alpha \zeta^2 \psi_{x,\theta} - \eta_3 h^2 \zeta \psi_{x,x\theta} \\
& + 2\eta_5 h^2 \cos^2 \alpha \zeta^2 \psi_{\theta} + 2\eta_5 h t \cos^2 \alpha \zeta^2 \psi'_{\theta} - \eta_{10} \zeta \psi'_{x,\theta} \\
& - \eta_{11} \zeta^2 \psi'_{\theta,\theta\theta} - \eta_{11} \sin \alpha \zeta^2 \psi'_{x,\theta} + 2\eta_{12} h^2 \sin^2 \alpha \zeta^2 \psi_{\theta} \\
& - 2\eta_{12} h^2 (\psi_{\theta,xx} + \sin \alpha \zeta \psi_{\theta,x}) - 2\eta_{12} h^2 \sin \alpha \zeta^2 \psi_{x,\theta} \\
& - 2\eta_{12} h^2 \zeta \psi_{x,x\theta} - \eta_{13} \sin \alpha \zeta^2 \psi'_{x,\theta} - \eta_{13} \zeta \psi'_{x,x\theta}
\end{aligned}$$

$$\begin{aligned}
& + \eta_{13} \sin^2 \alpha \zeta^2 \psi'_\theta - \eta_{13} (\psi'_{\theta,xx} + \sin \alpha \zeta \psi'_{\theta,x}) + 2\eta_{16} \psi_\theta \\
& + 2\eta_{16} \zeta w_{,\theta} - 2\eta_{16} \cos \alpha \zeta v + J\psi_{\theta,tt}] \delta \psi_\theta + [-2\eta_1 h^2 (\psi_{x,xx} \\
& + \sin \alpha \zeta \psi_{x,x}) + 2\eta_2 h^2 \sin^2 \alpha \zeta^2 \psi_x + 2\eta_2 h^2 \sin \alpha \zeta^2 \psi_{\theta,\theta} \\
& - \eta_3 h^2 \zeta \psi_{\theta,x\theta} - \eta_9 (\psi'_{x,xx} + \sin \alpha \zeta \psi'_{x,x}) - \eta_{10} \zeta \psi'_{\theta,x\theta} \\
& + \eta_{11} \sin \alpha \zeta^2 \psi'_{\theta,\theta} + \eta_{11} \sin^2 \alpha \zeta \psi'_x - 2\eta_{12} h^2 \zeta^2 \psi_{x,\theta\theta} \\
& + 2\eta_{12} h^2 \sin \alpha \zeta^2 \psi_{\theta,\theta} - 2\eta_{12} h^2 \zeta \psi_{\theta,x\theta} - \eta_{13} \zeta^2 \psi'_{x,\theta\theta} \\
& + \eta_{13} \sin \alpha \zeta^2 \psi'_{\theta,\theta} - \eta_{13} \zeta \psi'_{\theta,x\theta} + 2\eta_{15} \psi_x + 2\eta_{15} w_{,x} \\
& + J\psi_{x,tt}] \delta \psi_x \} \zeta^{-1} d\theta dx dt + \int_t \int_\theta \{ [2\eta_1 u_{,x} + \eta_3 \zeta v_{,\theta} \\
& + \eta_3 \sin \alpha \zeta u + \eta_3 \cos \alpha \zeta w] \delta u + [2\eta_{12} v_{,x} + 2\eta_{12} \zeta u_{,\theta} \\
& - 2\eta_{12} \sin \alpha \zeta v] \delta v + [+2\eta_4 w_{,x} + 2\eta_4 \psi'_x + 2\eta_{15} \psi_x \\
& + 2\eta_{15} w_{,x}] \delta w + [+ \eta_{13} \zeta \psi_{x,\theta} - \eta_{13} \sin \alpha \zeta \psi_\theta + \eta_{13} \psi_{\theta,x} \\
& + 2\eta_{14} \psi'_{\theta,x} + 2\eta_{14} \zeta \psi'_{x,\theta} - 2\eta_{14} \sin \alpha \zeta \psi'_\theta] \delta \psi'_\theta + [+2\eta_6 \psi'_{x,x} \\
& + \eta_8 \zeta \psi'_{\theta,\theta} + \eta_8 \sin \alpha \zeta \psi'_x + \eta_9 \psi_{x,x} + \eta_{10} \zeta \psi_{\theta,\theta} \\
& + \eta_{10} \sin \alpha \zeta \psi_x] \delta \psi'_x + [+2\eta_{12} h^2 \psi_{\theta,x} + 2\eta_{12} h^2 \zeta \psi_{x,\theta} \\
& - 2\eta_{12} h^2 \sin \alpha \zeta \psi_\theta + \eta_{13} \zeta \psi'_{x,\theta} - \eta_{13} \sin \alpha \zeta \psi'_\theta + \eta_{13} \psi'_{\theta,x} \delta \psi_\theta \\
& + [+2\eta_1 h^2 \psi_{x,x} + \eta_3 h^2 \zeta \psi_{\theta,\theta} + \eta_3 h^2 \sin \alpha \zeta \psi_x + \eta_9 \psi'_{x,x} \\
& + \eta_{10} \zeta \psi'_{\theta,\theta} + \eta_{10} \sin \alpha \zeta \psi'_x] \delta \psi_x \} \Big|_{x_1}^{x_2} \zeta^{-1} d\theta dt \tag{B-4}
\end{aligned}$$

The coefficients of the virtual displacements in the first half of Equation (B-4) are identified as the equations of motion, while those in the second half are identified with the boundary conditions.

The equations of motion are then written as:

$$\begin{aligned}
 & -2\eta_1(u_{,xx} + \sin \alpha \zeta u_{,x}) - 2\eta_{12}\zeta^2 u_{,\theta\theta} + 2\eta_2 \sin^2 \alpha \zeta^2 u \\
 & + 2(\eta_2 + \eta_{12}) \sin \alpha \zeta^2 v_{,\theta} - (2\eta_{12} + \eta_3)\zeta v_{,x\theta} + 2\eta_2 \sin \alpha \cos \alpha \zeta^2 w \\
 & - \eta_3 \cos \alpha \zeta w_{,x} + \bar{m}u_{,tt} = 0, \tag{B-5a}
 \end{aligned}$$

$$\begin{aligned}
 & -(2\eta_{12} + \eta_{13})\zeta u_{,x\theta} - 2 \sin \alpha (\eta_2 + \eta_{12})\zeta^2 u_{,\theta} - 2\eta_{12}(v_{,xx} \\
 & + \zeta \sin \alpha v_{,x}) + 2[\cos^2 \alpha (\eta_5 + \eta_{16}) + \sin^2 \alpha \eta_{12}]\zeta^2 v - 2\eta_2 \zeta^2 v_{,\theta\theta} \\
 & - 2[\eta_2 + \eta_5 + \eta_{16}] \cos \alpha \zeta^2 w_{,\theta} - 2\eta_5 \cos \alpha \zeta \psi'_{\theta} - 2\eta_{16} \cos \alpha \zeta \psi_{\theta} \\
 & + \bar{m}v_{,tt} = 0, \tag{B-5b}
 \end{aligned}$$

$$\begin{aligned}
 & \eta_3 \cos \alpha \zeta u_{,x} + 2\eta_2 \sin \alpha \cos \alpha \zeta^2 u + 2[\eta_2 + \eta_5 + \eta_{16}] \cos \alpha \zeta^2 v_{,\theta} \\
 & - 2[\eta_4 + \eta_{15}](w_{,xx} + \zeta \sin \alpha w_{,x}) - 2(\eta_5 + \eta_{16})\zeta^2 w_{,\theta\theta} \\
 & + 2\eta_2 \cos^2 \alpha \zeta^2 w - 2\eta_5 \zeta \psi'_{\theta,\theta} - 2\eta_4 (\psi'_{x,x} + \zeta \sin \alpha \psi'_x) - 2\eta_{16} \zeta \psi_{\theta,\theta} \\
 & - 2\eta_{15} (\psi_{x,x} + \zeta \sin \alpha \psi_x) + \bar{m}w_{,tt} = 0, \tag{B-5c}
 \end{aligned}$$

$$\begin{aligned}
 & -2\eta_5 \cos \alpha \zeta v + 2\eta_5 \zeta w_{,\theta} - 2\eta_{14} (\psi'_{\theta,xx} + \zeta \sin \alpha \psi'_{\theta,x}) \\
 & - 2\eta_7 \zeta^2 \psi'_{\theta,\theta\theta} + 2(\eta_5 + \eta_{14} \sin^2 \alpha \zeta^2) \psi'_{\theta} - (\eta_8 + 2\eta_{14}) \zeta \psi'_{x,x\theta} \\
 & - 2(\eta_7 + \eta_{14}) \sin \alpha \zeta^2 \psi'_{x,\theta} - \eta_{13} (\psi_{\theta,xx} + \zeta \sin \alpha \psi_{\theta,x}) \\
 & - \eta_{11} \zeta^2 \psi_{\theta,\theta\theta} + (2\eta_5 \cos^2 \alpha + \eta_{13} \sin^2 \alpha) \zeta^2 \psi_{\theta}
 \end{aligned}$$

$$- (\eta_{10} + \eta_{13}) \zeta \psi_{x,x\theta} - (\eta_{11} + \eta_{13}) \sin \alpha \zeta^2 \psi_{x,\theta} + 2J' \psi'_{\theta,tt} = 0, \quad (B-5d)$$

$$\begin{aligned} & 2\eta_{4w,x} - (\eta_8 + 2\eta_{14}) \zeta \psi'_{\theta,x\theta} + 2(\eta_7 + \eta_{14}) \sin \alpha \zeta^2 \psi'_{\theta,\theta} \\ & - 2\eta_6 (\psi'_{x,x} + \zeta \sin \alpha \psi'_x) + 2[\eta_4 + \eta_7 \sin^2 \alpha \zeta^2] \psi'_x - 2\eta_{14} \zeta^2 \psi'_{x,\theta\theta} \\ & - (\eta_{10} + \eta_{13}) \zeta \psi_{\theta,x\theta} + (\eta_{11} + \eta_{13}) \sin \alpha \zeta^2 \psi_{\theta,\theta} - \eta_9 (\psi_{x,xx} \\ & + \zeta \sin \alpha \psi_{x,x}) + \eta_{11} \zeta \sin^2 \alpha \psi_x - \eta_{13} \zeta^2 \psi_{x,\theta\theta} + 2J' \psi'_{x,tt} = 0, \quad (B-5e) \end{aligned}$$

$$\begin{aligned} & - 2\eta_{16} \cos \alpha \zeta v + 2\eta_{16} \zeta w_{,\theta} - \eta_{13} (\psi'_{\theta,xx} + \zeta \sin \alpha \psi'_{\theta,x}) \\ & + (2\eta_5 h t \cos^2 \alpha + \eta_{13} \sin^2 \alpha) \zeta^2 \psi'_{\theta} - \eta_{11} \zeta^2 \psi'_{\theta,\theta\theta} - (\eta_{10} + \eta_{13}) \zeta \psi'_{x,x\theta} \\ & - (\eta_{11} + \eta_{13}) \sin \alpha \zeta^2 \psi'_{x,\theta} - 2\eta_{12} h^2 (\psi_{\theta,xx} + \zeta \sin \alpha \psi_{\theta,x}) \\ & + 2(\eta_5 h^2 \cos^2 \alpha \zeta^2 + \eta_{16} + \eta_{12} h^2 \sin^2 \alpha \zeta^2) \psi_{\theta} - 2\eta_2 h^2 \zeta^2 \psi_{\theta,\theta\theta} \\ & - (\eta_3 h^2 + 2\eta_{12} h^2) \zeta \psi_{x,x\theta} - 2h^2 \sin \alpha (\eta_2 + \eta_{12}) \zeta^2 \psi_{x,\theta} + J \psi_{\theta,tt} = 0, \quad (B-5f) \end{aligned}$$

$$\begin{aligned} & + 2\eta_{15w,x} - (\eta_{10} + \eta_{13}) \zeta \psi'_{\theta,x\theta} + (\eta_{11} + \eta_{13}) \sin \alpha \zeta^2 \psi'_{\theta,\theta} \\ & - \eta_9 (\psi'_{x,xx} + \zeta \sin \alpha \psi'_{x,x}) + \eta_{11} \zeta \sin^2 \alpha \psi'_x - \eta_{13} \zeta^2 \psi'_{x,\theta\theta} \\ & - h^2 (\eta_3 + 2\eta_{12}) \zeta \psi_{\theta,x\theta} + 2h^2 \sin \alpha (\eta_2 + \eta_{12}) \zeta^2 \psi_{\theta,\theta} - 2\eta_1 h^2 (\psi_{x,xx} \\ & + \zeta \sin \alpha \psi_{x,x}) + 2(\eta_2 h^2 \sin^2 \alpha \zeta^2 + \eta_{15}) \psi_x - 2\eta_{12} h^2 \zeta^2 \psi_{x,\theta\theta} \\ & + J \psi_{x,tt} = 0. \quad (B5-g) \end{aligned}$$

The set of boundary conditions is

Either  $u = 0$ ,

$$\text{or } 2\eta_1 u_{,x} + \eta_3 (\sin \alpha \zeta u + \zeta v_{,\theta} + \cos \alpha \zeta w) = 0. \quad (B-6a)$$

Either  $v = 0$ ,

$$\text{or } 2\eta_{12} (\zeta u_{,\theta} + v_{,x} - \sin \alpha \zeta v) = 0. \quad (B-6b)$$

Either  $w = 0$ ,

$$\text{or } 2(\eta_4 + \eta_{15})w_{,x} + 2\eta_4\psi'_{x,x} + 2\eta_{15}\psi'_{x,x} = 0. \quad (\text{B-6c})$$

Either  $\psi'_\theta = 0$

$$\text{or } 2\eta_{14}(\psi'_{\theta,x} - \sin \alpha \zeta\psi'_\theta + \zeta\psi'_{x,\theta}) + \eta_{13}(\psi_{\theta,x} - \sin \alpha \zeta\psi_\theta + \zeta\psi_{x,\theta}) = 0. \quad (\text{B-6d})$$

Either  $\psi'_x = 0$ ,

$$\text{or } 2\eta_6\psi'_{x,x} + \eta_8(\sin \alpha \zeta\psi'_x + \zeta\psi'_{\theta,\theta}) + \eta_9\psi_{x,x} + \eta_{10}(\sin \alpha \zeta\psi_x + \zeta\psi_{\theta,\theta}) = 0. \quad (\text{B-6e})$$

Either  $\psi_\theta = 0$ ,

$$\text{or } \eta_{13}(\psi'_{\theta,x} - \sin \alpha \zeta\psi'_\theta + \zeta\psi'_{x,\theta}) + 2h^2\eta_{12}(\psi_{\theta,x} - \sin \alpha \zeta\psi_\theta + \zeta\psi_{x,\theta}) = 0. \quad (\text{B-6f})$$

Either  $\psi_x = 0$ ,

$$\text{or } \eta_9\psi'_{x,x} + \eta_{10}(\sin \alpha \zeta\psi'_x + \zeta\psi'_{\theta,\theta}) + 2h^2\eta_1\psi_{x,x} + h^2\eta_3(\sin \alpha \zeta\psi_x + \zeta\psi_{\theta,\theta}) = 0. \quad (\text{B-6g})$$

Each of the differential equations, given in Equations (B-6a) - (B-6g) as alternatives to the vanishing of displacements or rotations, expresses a force or moment. A rather simple integration will verify the following identifications.

Equation (B-6a) represents the normal force in the meridional direction,  $F_x$ , where

$$F_x = \int_{-h-2t}^{-h} \sigma_{xx}^i dz + \int_h^{h+2t} \sigma_{xx}^o dz, \quad (\text{B-7a})$$

$$F_x = \bar{E}'_x \int_{-h-2t}^{-h} (e_{xx}^i + \nu'_{\theta x} e_{\theta\theta}^i) dz + \bar{E}'_x \int_h^{h+2t} (e_{xx}^o + \nu'_{\theta x} e_{\theta\theta}^o) dz. \quad (B-7b)$$

Equation (B-6b) represents the in-plane shearing force,  $F_{x\theta}$ ,

where

$$F_{x\theta} = \int_{-h-2t}^{-h} \sigma_{x\theta}^i dz + \int_h^{h+2t} \sigma_{x\theta}^o dz, \quad (B-8a)$$

$$F_{x\theta} = G'_{x\theta} \int_{-h-2t}^{-h} e_{x\theta}^i dz + G'_{x\theta} \int_h^{h+2t} e_{x\theta}^o dz. \quad (B-8b)$$

Equation (B-6c) represents the transverse shear force,  $Q_x$ ,

where

$$Q_x = \int_{-h-2t}^{-h} \sigma_{zx}^i dz + \int_{-h}^h \sigma_{zx}^c dz + \int_h^{h+2t} \sigma_{zx}^o dz, \quad (B-9a)$$

$$Q_x = K'_x G'_{zx} \int_{-h-2t}^{-h} e_{zx}^i dz + K_x G_{zx} \int_{-h}^h e_{zx}^c dz + K'_x G'_{zx} \int_h^{h+2t} e_{zx}^o dz. \quad (B-9b)$$

Concerning Equations (B-6d) through (B-6g), one must recall the definitions of  $\psi'_\theta$ ,  $\psi'_x$ ,  $\psi_\theta$ , and  $\psi_x$ . It is then apparent that if  $\psi'_\theta$  is zero,  $\psi_\theta$  must also be zero. Likewise, if  $\psi'_x$  is zero,  $\psi_x$  must also be zero. Therefore, Equations (B-6d) and (B-6f) actually represent only one boundary condition. When added together, they represent the twisting moment  $M_{x\theta}$ , where

$$M_{x\theta} = \int_{-h-2t}^{-h} z \sigma_{x\theta}^i dz + \int_h^{h+2t} z \sigma_{x\theta}^o dz, \quad (B-10a)$$

$$M_{x\theta} = G'_{x\theta} \int_{-h-2t}^{-h} z e_{x\theta}^i dz + G'_{x\theta} \int_h^{h+2t} z e_{x\theta}^o dz. \quad (B-10b)$$

Similarly, when Equations (B-6e) and (B-6g) are added together, they represent the bending moment,  $M_x$ , where

$$M_x = \int_{-h-2t}^{-h} z \sigma_{xx}^i dz + \int_h^{h+2t} z \sigma_{xx}^o dz, \quad (B-11a)$$

$$M_x = \bar{E}'_x \int_{-h-2t}^{-h} z (e_{xx}^i + \nu'_{\theta x} e_{\theta\theta}^i) dz + \bar{E}'_x \int_h^{h+2t} z (e_{xx}^o + \nu'_{\theta x} e_{\theta\theta}^o) dz. \quad (B-11b)$$

The complete set of boundary conditions, at  $x=0$  and  $x=L$ , may now be given by

Either  $u = 0$ ,

$$\text{or } F_x = 2\eta_1 u_{,x} + \eta_3 (\sin \alpha \zeta u + \zeta v_{,\theta} + \cos \alpha \zeta w) = 0. \quad (B-12a)$$

Either  $v = 0$ ,

$$\text{or } F_{x\theta} = 2\eta_{12} (\zeta u_{,\theta} + v_{,x} - \sin \alpha \zeta v) = 0. \quad (B-12b)$$

Either  $w = 0$ ,

$$\text{or } Q_x = 2(\eta_4 + \eta_{15}) w_{,x} + 2\eta_4 \psi'_{,x} + 2\eta_{15} \psi_x = 0. \quad (B-12c)$$

Either  $\psi'_{\theta} = \psi_{\theta} = 0$ ,

$$\text{or } M_{x\theta} = (2\eta_{14} + \eta_{13}) (\psi'_{\theta,x} - \sin \alpha \zeta \psi'_{x,\theta}) + \eta_{13} + 2h^2 \eta_{12} (\psi_{\theta,x} - \sin \alpha \zeta \psi_{\theta} + \zeta \psi_{x,\theta}) = 0. \quad (B-12d)$$

Either  $\psi'_{,x} = \psi_x = 0$ ,

$$\text{or } M_x = (2\eta_6 + \eta_9) \psi'_{x,x} + (\eta_8 + \eta_{10}) (\sin \alpha \zeta \psi'_{,x} + \zeta \psi'_{\theta,\theta}) + (\eta_9 + 2h^2 \eta_1) \psi_{x,x} + (\eta_{10} + h^2 \eta_3) (\sin \alpha \zeta \psi_x + \zeta \psi_{\theta,\theta}) = 0. \quad (B-12e)$$

## APPENDIX C

IDENTIFICATION OF INTEGRAL FORMS

The insertion of Equations (2-11) into Equations (2-12) gives a definite set of eighty integral forms. For convenience in handling of these integrals, each will be assigned an indicative name. Some of the integrals (forty-five of them) are not independent, but are identical to other integrals. Those integrals which need not be recalculated are followed in parentheses by the name of the integral to which they are identical. In particular, it is noted that the integrals involving  $\psi_{\theta}(\varphi_6)$  and  $\psi_x(\varphi_7)$  need not be recalculated, since the series assumed for them are identical to  $\psi_{\theta}'(\varphi_4)$  and  $\psi_x'(\varphi_5)$ , respectively. It should be kept in mind that the value of each of the integrals depends on the values of the subscripts k and m. The notation  $k \rightarrow m$  indicates the reversal of the roles of k and m.

$$\text{IR111} \equiv \int_0^1 R^{-2} \varphi_{1m} \varphi_{1k} d\epsilon$$

$$\text{IR11} \equiv \int_0^1 \varphi_{1m} \varphi_{1k} d\epsilon$$

$$\text{IE11} \equiv \int_0^1 R^{-1} \varphi_{1m, \epsilon} \varphi_{1k} d\epsilon$$

$$\text{IREE11} \equiv \int_0^1 \varphi_{1m, \epsilon \epsilon} \varphi_{1k} d\epsilon$$

$$\text{IR121} \equiv \int_0^1 R^{-2} \varphi_{2m} \varphi_{1k} d\epsilon$$

$$\text{IE21} \equiv \int_0^1 R^{-1} \varphi_{2m, \epsilon} \varphi_{1k} d\epsilon$$

$$\text{IR131} \equiv \int_0^1 R^{-2} \varphi_{3m} \varphi_{1k} d\epsilon$$

$$\text{IE31} \equiv \int_0^1 R^{-1} \varphi_{3m, \epsilon} \varphi_{1k} d\epsilon$$

$$\text{IR112} \equiv \int_0^1 R^{-2} \varphi_{1m} \varphi_{2k} d\epsilon$$

(IR121,  $k \rightarrow m$ )

$$\text{IE12} \equiv \int_0^1 R^{-1} \varphi_{1m, \epsilon} \varphi_{2k} d\epsilon$$

$$\text{IR122} \equiv \int_0^1 R^{-2} \varphi_{2m} \varphi_{2k} d\epsilon$$

$$\text{IR22} \equiv \int_0^1 \varphi_{2m} \varphi_{2k} d\epsilon$$

$$\text{IE22} \equiv \int_0^1 R^{-1} \varphi_{2m, \epsilon} \varphi_{2k} d\epsilon$$

$$\text{IREE22} \equiv \int_0^1 \varphi_{2m, \epsilon} \varphi_{2k} d\epsilon$$

$$\text{IR132} \equiv \int_0^1 R^{-2} \varphi_{3m} \varphi_{2k} d\epsilon$$

$$\text{I42} \equiv \int_0^1 R^{-1} \varphi_{4m} \varphi_{2k} d\epsilon$$

$$\text{I62} \equiv \int_0^1 R^{-1} \varphi_{6m} \varphi_{2k} d\epsilon$$

(I42)

$$\text{IR113} \equiv \int_0^1 R^{-2} \varphi_{1m} \varphi_{3k} d\epsilon \quad (\text{IR131, } k \rightarrow m)$$

$$\text{IE13} \equiv \int_0^1 R^{-1} \varphi_{1m, \epsilon} \varphi_{3k} d\epsilon$$

$$\text{IR123} \equiv \int_0^1 R^{-2} \varphi_{2m} \varphi_{3k} d\epsilon \quad (\text{IR132, } k \rightarrow m)$$

$$\text{IR133} \equiv \int_0^1 R^{-2} \varphi_{3m} \varphi_{3k} d\epsilon$$

$$\text{IR33} \equiv \int_0^1 \varphi_{3m} \varphi_{3k} d\epsilon$$

$$\text{IE33} \equiv \int_0^1 R^{-1} \varphi_{3m, \epsilon} \varphi_{3k} d\epsilon$$

$$\text{IREE33} \equiv \int_0^1 \varphi_{3m, \epsilon \epsilon} \varphi_{3k} d\epsilon$$

$$\text{I43} \equiv \int_0^1 R^{-1} \varphi_{4m} \varphi_{3k} d\epsilon$$

$$\text{I53} \equiv \int_0^1 R^{-1} \varphi_{5m} \varphi_{3k} d\epsilon$$

$$\text{IRE53} \equiv \int_0^1 \varphi_{5m, \epsilon} \varphi_{3k} d\epsilon$$

$$\text{I63} \equiv \int_0^1 R^{-1} \varphi_{6m} \varphi_{3k} d\epsilon \quad (\text{I43})$$

$$\text{I73} \equiv \int_0^1 R^{-1} \varphi_{7m} \varphi_{3k} d\epsilon \quad (\text{I53})$$

$$\text{IRE73} \equiv \int_0^1 \varphi_{7m, \epsilon} \varphi_{3k} d\epsilon \quad (\text{IRE53})$$

$$I24 \equiv \int_0^1 R^{-1} \varphi_{2m} \varphi_{4k} d\epsilon \quad (I42, k \rightarrow m)$$

$$I34 \equiv \int_0^1 R^{-1} \varphi_{3m} \varphi_{4k} d\epsilon \quad (I43, k \rightarrow m)$$

$$IR144 \equiv \int_0^1 R^{-2} \varphi_{4m} \varphi_{4k} d\epsilon$$

$$IR44 \equiv \int_0^1 \varphi_{4m} \varphi_{4k} d\epsilon$$

$$IE44 \equiv \int_0^1 R^{-1} \varphi_{4m, \epsilon} \varphi_{4k} d\epsilon$$

$$IREE44 \equiv \int_0^1 \varphi_{4m, \epsilon \epsilon} \varphi_{4k} d\epsilon$$

$$IR154 \equiv \int_0^1 R^{-2} \varphi_{5m} \varphi_{4k} d\epsilon$$

$$IE54 \equiv \int_0^1 R^{-1} \varphi_{5m, \epsilon} \varphi_{4k} d\epsilon$$

$$IR164 \equiv \int_0^1 R^{-2} \varphi_{6m} \varphi_{4k} d\epsilon \quad (IR144)$$

$$IE64 \equiv \int_0^1 R^{-1} \varphi_{6m, \epsilon} \varphi_{4k} d\epsilon \quad (IE44)$$

$$IREE64 \equiv \int_0^1 \varphi_{6m, \epsilon \epsilon} \varphi_{4k} d\epsilon \quad (IREE44)$$

$$IR174 \equiv \int_0^1 R^{-2} \varphi_{7m} \varphi_{4k} d\epsilon \quad (IR154)$$

$$IE74 \equiv \int_0^1 R^{-1} \varphi_{7m, \epsilon} \varphi_{4k} d\epsilon \quad (IE54)$$

$$\text{IRE35} \equiv \int_0^1 \varphi_{3m, \epsilon} \varphi_{5k} \, d\epsilon$$

$$\text{IR145} \equiv \int_0^1 R^{-2} \varphi_{4m} \varphi_{5k} \, d\epsilon \quad (\text{IR154, } k \rightarrow m)$$

$$\text{IE45} \equiv \int_0^1 R^{-1} \varphi_{4m, \epsilon} \varphi_{5k} \, d\epsilon$$

$$\text{IR155} \equiv \int_0^1 R^{-2} \varphi_{5m} \varphi_{5k} \, d\epsilon$$

$$\text{IR55} \equiv \int_0^1 \varphi_{5m} \varphi_{5k} \, d\epsilon$$

$$\text{IE55} \equiv \int_0^1 R^{-1} \varphi_{5m, \epsilon} \varphi_{5k} \, d\epsilon$$

$$\text{IREE55} \equiv \int_0^1 \varphi_{5m, \epsilon \epsilon} \varphi_{5k} \, d\epsilon$$

$$\text{IR165} \equiv \int_0^1 R^{-2} \varphi_{6m} \varphi_{5k} \, d\epsilon \quad (\text{IR154, } k \rightarrow m)$$

$$\text{IE65} \equiv \int_0^1 R^{-1} \varphi_{6m, \epsilon} \varphi_{5k} \, d\epsilon \quad (\text{IE45})$$

$$\text{IR175} \equiv \int_0^1 R^{-2} \varphi_{7m} \varphi_{5k} \, d\epsilon \quad (\text{IR155})$$

$$\text{IE75} \equiv \int_0^1 R^{-1} \varphi_{7m, \epsilon} \varphi_{5k} \, d\epsilon \quad (\text{IE55})$$

$$\text{IREE75} \equiv \int_0^1 \varphi_{7m, \epsilon \epsilon} \varphi_{5k} \, d\epsilon \quad (\text{IREE55})$$

$$\text{I26} \equiv \int_0^1 R^{-1} \varphi_{2m} \varphi_{6k} \, d\epsilon \quad (\text{I42, } k \rightarrow m)$$

$$I36 \equiv \int_0^1 R^{-1} \varphi_{3m} \varphi_{6k} ds \quad (I43, k \rightarrow m)$$

$$IR146 \equiv \int_0^1 R^{-2} \varphi_{4m} \varphi_{6k} ds \quad (IR144)$$

$$IE46 \equiv \int_0^1 R^{-1} \varphi_{4m, \epsilon} \varphi_{6k} ds \quad (IE44)$$

$$IREE46 \equiv \int_0^1 \varphi_{4m, \epsilon \epsilon} \varphi_{6k} ds \quad (IREE44)$$

$$IR156 \equiv \int_0^1 R^{-2} \varphi_{5m} \varphi_{6k} ds \quad (IR154)$$

$$IE56 \equiv \int_0^1 R^{-1} \varphi_{5m, \epsilon} \varphi_{6k} ds \quad (IE54)$$

$$IR166 \equiv \int_0^1 R^{-2} \varphi_{6m} \varphi_{6k} ds \quad (IR144)$$

$$IR66 \equiv \int_0^1 \varphi_{6m} \varphi_{6k} ds \quad (IR44)$$

$$IE66 \equiv \int_0^1 R^{-1} \varphi_{6m, \epsilon} \varphi_{6k} ds \quad (IE44)$$

$$IREE66 \equiv \int_0^1 \varphi_{6m, \epsilon \epsilon} \varphi_{6k} ds \quad (IREE44)$$

$$IR176 \equiv \int_0^1 R^{-2} \varphi_{7m} \varphi_{6k} ds \quad (IR154)$$

$$IE76 \equiv \int_0^1 R^{-1} \varphi_{7m, \epsilon} \varphi_{6k} ds \quad (IE54)$$

$$IRE37 \equiv \int_0^1 \varphi_{3m, \epsilon} \varphi_{7k} ds \quad (IRE35)$$

$$\text{IR147} \equiv \int_0^1 R^{-2} \varphi_{4m} \varphi_{7k} \, d\epsilon \quad (\text{IR154, } k \rightarrow m)$$

$$\text{IE47} \equiv \int_0^1 R^{-1} \varphi_{4m, \epsilon} \varphi_{7k} \, d\epsilon \quad (\text{IE45})$$

$$\text{IR157} \equiv \int_0^1 R^{-2} \varphi_{5m} \varphi_{7k} \, d\epsilon \quad (\text{IR155})$$

$$\text{IE57} \equiv \int_0^1 R^{-1} \varphi_{5m, \epsilon} \varphi_{7k} \, d\epsilon \quad (\text{IE55})$$

$$\text{IREE57} \equiv \int_0^1 \varphi_{5m, \epsilon \epsilon} \varphi_{7k} \, d\epsilon \quad (\text{IREE55})$$

$$\text{IR167} \equiv \int_0^1 R^{-2} \varphi_{6m} \varphi_{7k} \, d\epsilon \quad (\text{IR154, } k \rightarrow m)$$

$$\text{IE67} \equiv \int_0^1 R^{-1} \varphi_{6m, \epsilon} \varphi_{7k} \, d\epsilon \quad (\text{IE45})$$

$$\text{IR177} \equiv \int_0^1 R^{-2} \varphi_{7m} \varphi_{7k} \, d\epsilon \quad (\text{IR155})$$

$$\text{IR77} \equiv \int_0^1 \varphi_{7m} \varphi_{7k} \, d\epsilon \quad (\text{IR55})$$

$$\text{IE77} \equiv \int_0^1 R^{-1} \varphi_{7m, \epsilon} \varphi_{7k} \, d\epsilon \quad (\text{IE55})$$

$$\text{IREE77} \equiv \int_0^1 \varphi_{7m, \epsilon \epsilon} \varphi_{7k} \, d\epsilon \quad (\text{IREE55})$$

## APPENDIX D

### EVALUATION OF INTEGRALS FOR FREELY SUPPORTED BOUNDARY CONDITION

Some of the integrals reduce to the form  $\int (\sin x/x) dx$  or  $\int (\cos x/x) dx$ , for which IBM has a standard algorithm, SICI. The description and usage of the subroutine is given in Appendix G.

For some of the integrals, a closed-form solution could not be found. In these cases, a simple trapezoidal numerical integration subroutine, QTFE, was used. This routine is also described in Appendix G.

In the following pages, the algorithm for each of the thirty-five independent integrals is given. The actual computer code is presented in Appendix H.

$$IR111 = \int_0^1 R^{-2-2\nu'_0 x} \cos m\pi \epsilon \cos k\pi \epsilon d\epsilon \quad (D-1)$$

If  $\sin \alpha = 0$ ,

$$IR111 = \bar{R}_0^{-2-2\nu'_0} \int_0^1 \cos m\pi \epsilon \cos k\pi \epsilon d\epsilon$$

$$= \begin{cases} \frac{1}{2} \bar{R}_0^{-2-2\nu'_0} & ; m=k \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$IR11 = \int_0^1 R^{-2\nu'_{\theta x}} \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (D-2)$$

If  $\sin \alpha = 0$ ,

$$\begin{aligned} IR11 &= \bar{R}_o^{-2\nu'_{\theta x}} \int_0^1 \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \\ &= \begin{cases} \frac{1}{2} \bar{R}_o^{-2\nu'_{\theta x}} & ; m=k \\ 0 & ; \text{otherwise} \end{cases} \end{aligned}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\begin{aligned} IE11 &= -\nu'_{\theta x} \sin \alpha \int_0^1 R^{-2-2\nu'_{\theta x}} \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \\ &\quad - m\pi \int_0^1 R^{-1-2\nu'_{\theta x}} \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (D-3) \end{aligned}$$

If  $\sin \alpha = 0$ ,

$$\begin{aligned} IE11 &= -m\pi \bar{R}_o^{-1-2\nu'_{\theta x}} \int_0^1 \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \\ &= \begin{cases} \frac{-2m^2 \bar{R}_o^{-1-2\nu'_{\theta x}}}{(m^2 - k^2)} & ; m+k = \text{odd}, m \neq 0 \\ 0 & ; \text{otherwise.} \end{cases} \end{aligned}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\begin{aligned} IREE11 &= \int_0^1 [\nu'_{\theta x} (1 + \nu'_{\theta x}) \sin^2 \alpha R^{-2-2\nu'_{\theta x}} \\ &\quad - m^2 \pi^2] \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \\ &\quad + \int_0^1 2m\pi \nu'_{\theta x} \sin \alpha R^{-1-2\nu'_{\theta x}} \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (D-4) \end{aligned}$$

Combination of the three previous algorithms can be performed to give

$$\begin{aligned} \text{IREE11} &= v_{\theta x}' \sin \alpha [1 - v_{\theta x}' \sin \alpha] \text{IR111} - m^2 \pi^2 \text{IR11} \\ &\quad - 2v_{\theta x}' \sin \alpha \text{IE11} \end{aligned}$$

$$\text{IR121} = \int_0^1 R^{-2-2v_{\theta x}'} \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (\text{D-5})$$

If  $\sin \alpha = 0$ ,

$$\begin{aligned} \text{IR121} &= \bar{R}_0^{-2-v_{\theta x}'} \int_0^1 \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \\ &= \begin{cases} \frac{2m\bar{R}_0^{-2-v_{\theta x}'}}{\pi(m^2-k^2)} & ; m+k = \text{odd}, m \neq 0. \\ 0 & ; \text{otherwise.} \end{cases} \end{aligned}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\text{IE21} = m\pi \int_0^1 R^{-1-v_{\theta x}'} \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (\text{D-6})$$

If  $\sin \alpha = 0$ ,

$$\begin{aligned} \text{IE21} &= m\pi \bar{R}_0^{-1-v_{\theta x}'} \int_0^1 \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \\ &= \begin{cases} \frac{m\pi}{2} \bar{R}_0^{-1-v_{\theta x}'} & ; m=k. \\ 0 & ; \text{otherwise.} \end{cases} \end{aligned}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\text{IR131} = \text{IR121} \quad (\text{D-7})$$

$$\text{IE31} = \text{IE21} \quad (\text{D-8})$$

$$\begin{aligned}
 IE12 &= -v'_{\theta x} \sin \alpha \int_0^1 R^{-2-v'_{\theta x}} \cos m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \\
 &\quad - m\pi \int_0^1 R^{-1-v'_{\theta x}} \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon
 \end{aligned} \tag{D-9}$$

If  $\sin \alpha = 0$ ,

$$\begin{aligned}
 IE12 &= -m\pi \bar{R}_o^{-1-v'_{\theta x}} \int_0^1 \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \\
 &= \begin{cases} -\frac{1}{2} m\pi \bar{R}_o^{-1-v'_{\theta x}} & ; m=k. \\ 0 & ; \text{otherwise} \end{cases}
 \end{aligned}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$IR122 = \int_0^1 R^{-2} \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \tag{D-10}$$

If  $\sin \alpha = 0$ ,

$$\begin{aligned}
 IR122 &= \bar{R}_o^{-2} \int_0^1 \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \\
 &= \begin{cases} \frac{1}{2} \bar{R}_o^{-2} & ; m=k \\ 0 & ; \text{otherwise.} \end{cases}
 \end{aligned}$$

If  $\sin \alpha \neq 0$ , integrating Equation (D-10) by parts, one obtains

$$\begin{aligned}
 IR122 &= -(R \sin \alpha)^{-1} \sin m\pi\epsilon \sin k\pi\epsilon \Big|_0^1 + \int_0^1 (k\pi \sin m\pi\epsilon \cos k\pi\epsilon \\
 &\quad + m\pi \cos m\pi\epsilon \sin k\pi\epsilon) (d\epsilon/R \sin \alpha) \\
 &= \frac{1}{2 \sin \alpha} \int_0^1 k\pi [\sin(m+k)\pi\epsilon + \sin(m-k)\pi\epsilon] + m\pi [\sin(m+k)\pi\epsilon \\
 &\quad - \sin(m-k)\pi\epsilon] (d\epsilon/R)
 \end{aligned}$$

$$= \frac{\pi}{2 \sin \alpha} \int_0^1 (m+k) \sin (m+k)\pi e - (m-k) \sin (m-k)\pi e (de/R)$$

Letting the dummy variable  $u = \bar{R}_0 + e \sin \alpha$ , then

$$\begin{aligned} IR_{122} = \frac{\pi}{2 \sin \alpha} \int_{\bar{R}_0}^{\bar{R}_0 + \sin \alpha} & (m+k) \sin \left[ (m+k)\pi \left( \frac{u - \bar{R}_0}{\sin \alpha} \right) \right] \\ & - (m-k) \sin \left[ (m-k)\pi \frac{u - \bar{R}_0}{\sin \alpha} \right] \frac{du}{u \sin \alpha} \end{aligned}$$

Now letting dummy variable  $v = u/\sin \alpha$ , and letting  $\bar{\rho} = \bar{R}_0/\sin \alpha$ ,

$$\begin{aligned} IR_{122} = \frac{\pi}{2 \sin^2 \alpha} \int_{\bar{\rho}}^{\bar{\rho} + 1} & (m+k) \sin \left[ (m+k)\pi (v - \bar{\rho}) \right] \\ & - (m-k) \sin \left[ (m-k)\pi (v - \bar{\rho}) \right] \frac{dv}{v} \end{aligned}$$

Using the relation,

$$\sin (x-y) = \sin x \cos y - \cos x \sin y,$$

the preceding equation becomes

$$\begin{aligned} IR_{122} = \frac{\pi}{2 \sin^2 \alpha} \int_{\bar{\rho}}^{\bar{\rho} + 1} & (m+k) \left[ \sin (m+k)\pi v \cos (m+k)\pi \bar{\rho} \right. \\ & \left. - \cos (m+k)\pi v \sin (m+k)\pi \bar{\rho} \right] \frac{dv}{v} \\ & - \frac{\pi}{2 \sin^2 \alpha} \int_{\bar{\rho}}^{\bar{\rho} + 1} (m-k) \left[ \sin (m-k)\pi v \cos (m-k)\pi \bar{\rho} \right. \\ & \left. - \cos (m-k)\pi v \sin (m-k)\pi \bar{\rho} \right] \frac{dv}{v} \end{aligned}$$

The following symbols are now defined:

$$C_s, C_d = \cos (m \pm k)\pi \bar{\rho}$$

$$S_s, S_d = \sin (m \pm k)\pi \bar{\rho}$$

$$W_s, W_d = (m \pm k)\pi v$$

$$W_{si}, W_{di} = (m \pm k)\pi \bar{\rho}$$

$$W_{sf}, W_{df} = (m \pm k)\pi(\bar{\rho} + 1)$$

Now IR122 can be written as

$$\begin{aligned} \text{IR122} = & \frac{\pi(m+k)}{2 \sin^2 \alpha} \left[ C_s \int_{W_{si}}^{W_{sf}} \frac{\sin(W_s)}{W_s} dW_s - S_s \int_{W_{si}}^{W_{sf}} \frac{\cos(W_s)}{W_s} dW_s \right] \\ & - \frac{\pi(m-k)}{2 \sin^2 \alpha} \left[ C_d \int_{W_{di}}^{W_{df}} \frac{\sin(W_d)}{W_d} dW_d - S_d \int_{W_{di}}^{W_{df}} \frac{\cos(W_d)}{W_d} dW_d \right] \end{aligned}$$

Now the subroutine SICI may be used. It is noted that if  $m=k$ , the terms involving  $C_d$  and  $S_d$  reduce to zero.

$$\begin{aligned} \text{IR22} &= \int_0^1 \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon & (\text{D-11}) \\ &= \begin{cases} \frac{1}{2} & ; m=k, m \neq 0 \\ 0 & ; \text{otherwise} \end{cases} \end{aligned}$$

$$\text{IE22} = m\pi \int_0^1 R^{-1} \cos m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (\text{D-12})$$

If  $\sin \alpha = 0$ ,

$$\text{IE22} = \begin{cases} \frac{2km\bar{R}_0^{-1}}{(k^2 - m^2)} & ; m+k = \text{odd} \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ ,

$$\text{IE22} = \frac{m\pi}{2} \int_0^1 \frac{\sin(m+k)\pi\epsilon - \sin(m-k)\pi\epsilon}{R} \, d\epsilon$$

Using a procedure similar to the solution of IR122, and the same definitions,

$$IE22 = \frac{m\pi}{2 \sin \alpha} \left[ C_s \int_{W_{si}}^{W_{sf}} \frac{\sin(W_s)}{W_s} dW_s - S_s \int_{W_{si}}^{W_{sf}} \frac{\cos(W_s)}{W_s} dW_s \right] \\ - \frac{m\pi}{2 \sin \alpha} \left[ C_d \int_{W_{di}}^{W_{df}} \frac{\sin(W_d)}{W_d} dW_d - S_d \int_{W_{di}}^{W_{df}} \frac{\cos(W_d)}{W_d} dW_d \right]$$

When  $m=k$ , the  $C_d$  and  $S_d$  terms are zero.

$$IREE22 = -m^2 \pi^2 \int_0^1 \sin m\pi e \sin k\pi e de \quad (D-13)$$

$$= -m^2 \pi^2 IR22$$

$$IR132 = IR122 \quad (D-14)$$

$$I42 = \int_0^1 R^{-1} \sin m\pi e \sin k\pi e de \quad (D-15)$$

If  $\sin \alpha = 0$ ,

$$I42 = \begin{cases} \frac{1}{2} \bar{R}_0^{-1} & ; m=k, m \neq 0 \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ ,

$$I42 = \frac{1}{2} \int_0^1 \frac{\cos(m-k)\pi e - \cos(m+k)\pi e}{R} de$$

Following the procedure of IR122,

$$I42 = \frac{1}{2 \sin \alpha} \left[ C_d \int_{W_{di}}^{W_{df}} \frac{\cos(W_d)}{W_d} dW_d + S_d \int_{W_{di}}^{W_{df}} \frac{\sin(W_d)}{W_d} dW_d \right]$$

$$- \frac{1}{2 \sin \alpha} \left[ C_s \int_{W_{si}}^W \frac{W_s f \cos(W_s)}{W_s} dW_s + S_s \int_{W_{si}}^W \frac{W_s f \sin(W_s)}{W_s} dW_s \right]$$

Now SICI is used. If  $m=k$ , the terms involving  $C_d$  and  $S_d$  are replaced by

$$\left\{ \ln \left[ (\bar{R}_o + \sin \alpha) / \bar{R}_o \right] \right\} / (2 \sin \alpha).$$

$$IE13 = IE12 \quad (D-16)$$

$$IR133 = IR122 \quad (D-17)$$

$$IR33 = IR22 \quad (D-18)$$

$$IE33 = IE22 \quad (D-19)$$

$$IREE33 = IREE22 \quad (D-20)$$

$$I43 = I42 \quad (D-21)$$

$$I53 = \int_0^1 R^{-1-\nu'_{\theta x}} \cos m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (D-22)$$

If  $\sin \alpha = 0$ ,

$$I53 = \begin{cases} \frac{2k\bar{R}_o^{-1-\nu'_{\theta x}}}{\pi(k^2 - m^2)} & ; \quad m+k = \text{odd}, k \neq 0 \\ 0 & ; \quad \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\begin{aligned} IRE53 &= -\nu'_{\theta x} \sin \alpha \int_0^1 \bar{R}^{-1-\nu'_{\theta x}} \cos m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \\ &\quad - m\pi \int_0^1 R^{-\nu'_{\theta x}} \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \end{aligned} \quad (D-23)$$

If  $\sin \alpha = 0$ ,

$$\text{IRE53} = \begin{cases} -\frac{m\pi}{2} \bar{R}_0^{-\nu} \theta_x & ; m=k \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\text{IR144} = \text{IR122} \quad (\text{D-24})$$

$$\text{IR44} = \text{IR22} \quad (\text{D-25})$$

$$\text{IE44} = \text{IE22} \quad (\text{D-26})$$

$$\text{IREE44} = \text{IREE22} \quad (\text{D-27})$$

$$\text{IR154} = \text{IR121} \quad (\text{D-28})$$

$$\text{IE54} = \text{IE12} \quad (\text{D-29})$$

$$\text{IRE35} = m\pi \int_0^1 R^{-\nu} \theta_x \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (\text{D-30})$$

If  $\sin \alpha = 0$ ,

$$\text{IRE35} = \begin{cases} \frac{m\pi}{2} \bar{R}_0^{-\nu} \theta_x & ; m=k \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , use QTFE.

$$\text{IE45} = \text{IE21} \quad (\text{D-31})$$

$$\text{IR155} = \text{IR111} \quad (\text{D-32})$$

$$\text{IR55} = \text{IR11} \quad (\text{D-33})$$

$$\text{IE55} = \text{IE11} \quad (\text{D-34})$$

$$\text{IREE55} = \text{IREE11} \quad (\text{D-35})$$

## APPENDIX E

### EVALUATION OF INTEGRALS FOR CLAMPED-CLAMPED BOUNDARY CONDITION

Some of the integrals have been evaluated previously for the freely supported case. These will be noted as they occur.

$$IR111 = \int_0^1 R^{-2} \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (E-1)$$

See Equation (D-10).

$$IR11 = \int_0^1 \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (E-2)$$

See Equation (D-11).

$$IE11 = m\pi \int_0^1 R^{-1} \cos m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (E-3)$$

See Equation (D-12).

$$IREE11 = -m^2 \pi^2 \int_0^1 \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (E-4)$$

$$= -m^2 \pi^2 IR11$$

$$IR121 = IR111 \quad (E-5)$$

$$IE21 = IE11 \quad (E-6)$$

$$IR131 = IR111 \quad (E-7)$$

$$IE31 = IE11 \quad (E-8)$$

$$IE12 = IE11 \quad (E-9)$$

$$IR122 = IR111 \quad (E-10)$$

$$IR22 = IR11 \quad (E-11)$$

$$IE22 = IE11 \quad (E-12)$$

$$IREE22 = IREE11 \quad (E-13)$$

$$IR132 = IR111 \quad (E-14)$$

$$I42 = I53 \quad (E-15)$$

$$IE13 = IE11 \quad (E-16)$$

$$IR133 = IR111 \quad (E-17)$$

$$IR33 = IR11 \quad (E-18)$$

$$IE33 = IE11 \quad (E-19)$$

$$IREE33 = IREE11 \quad (E-20)$$

$$I43 = I42 \quad (E-21)$$

$$I53 = \int_0^1 R^{-1} \sin m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (E-22)$$

See Equation (D-15).

$$IRE53 = m\pi \int_0^1 \cos m\pi\epsilon \sin k\pi\epsilon \, d\epsilon \quad (E-23)$$

$$= \begin{cases} \frac{2km}{(k^2 - m^2)} & ; m+k = \text{odd} \\ 0 & ; \text{otherwise} \end{cases}$$

$$IR144 = IR111 \quad (E-24)$$

$$IR44 = IR11 \quad (E-25)$$

$$IE44 = IE11 \quad (E-26)$$

$$IREE44 = IREE11 \quad (E-27)$$

$$IR154 = IR111 \quad (E-28)$$

$$IE54 = IE11 \quad (E-29)$$

IRE35	=	IRE53	(E-30)
IE45	=	IE11	(E-31)
IR155	=	IR111	(E-32)
IR55	=	IR11	(E-33)
IE55	=	IE11	(E-34)
IREE55	=	IREE11	(E-35)

---

APPENDIX F

EVALUATION OF INTEGRALS FOR FREE-FREE

BOUNDARY CONDITION

Some of the integrals have been evaluated previously in Appendices D and E.

$$IR_{111} = \int_0^1 R^{-2} \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (F-1)$$

If  $\sin \alpha = 0$ ,

$$IR_{111} = \begin{cases} \bar{R}_0^{-2} & ; m=k=0 \\ \frac{1}{2}\bar{R}_0^{-2} & ; m=k \neq 0 \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , and  $m=k=0$ ,

$$IR_{111} = -\frac{1}{\sin \alpha} \left[ \frac{1}{\bar{R}_0 + \sin \alpha} - \frac{1}{\bar{R}_0} \right]$$

For the general case of  $\sin \alpha \neq 0$ ,  $m \neq k$ , integrating Equations (F-1) by parts, one obtains:

$$IR_{111} = \frac{-\cos m\pi\epsilon \cos k\pi\epsilon}{R \sin \alpha} \Big|_0^1 - \int_0^1 (k\pi \cos m\pi\epsilon \sin k\pi\epsilon + m\pi \sin m\pi\epsilon \cos k\pi\epsilon) (d\epsilon/R \sin \alpha)$$

Following the transformation of variables procedure for the freely

supported IR122, Equation (D-10),

$$\begin{aligned}
 \text{IR111} = & - \frac{\cos m\pi\epsilon \cos k\pi\epsilon}{R \sin \alpha} \Big|_0^1 - \frac{\pi(m+k)}{2 \sin^2 \alpha} \left[ C_s \int_{W_{si}}^{W_{sf}} \frac{\sin(W_s)}{W_s} dW_s \right. \\
 & - S_s \int_{W_{si}}^{W_{sf}} \frac{\cos(W_s)}{W_s} dW_s \Big] - \frac{\pi(m-k)}{2 \sin^2 \alpha} \left[ C_d \int_{W_{di}}^{W_{df}} \frac{\sin(W_d)}{W_d} dW_d \right. \\
 & \left. - S_d \int_{W_{di}}^{W_{df}} \frac{\cos(W_d)}{W_d} dW_d \right]
 \end{aligned}$$

Subroutine SICI may now be used. If  $m=k \neq 0$ , the terms involving  $C_d$  and  $S_d$  reduce to zero.

$$\begin{aligned}
 \text{IR11} &= \int_0^1 \cos m\pi\epsilon \cos k\pi\epsilon d\epsilon & (\text{F-2}) \\
 &= \begin{cases} 1 & ; m=k=0 \\ \frac{1}{2} & ; m=k \neq 0 \\ 0 & ; \text{otherwise} \end{cases}
 \end{aligned}$$

$$\text{IE11} = -m\pi \int_0^1 R^{-1} \sin m\pi\epsilon \cos k\pi\epsilon d\epsilon \quad (\text{F-3})$$

If  $m=0$ ,  $\text{IE11} = 0$ .

If  $\sin \alpha = 0$ ,

$$\text{IE11} = \begin{cases} \frac{-2m^2 \bar{R}_o^{-1}}{(m^2 - k^2)} & ; m+k = \text{odd} \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ ,

$$\text{IE11} = - \frac{m\pi}{2} \int_0^1 \frac{\sin(m+k)\pi\epsilon + \sin(m-k)\pi\epsilon}{R} d\epsilon$$

Using the transformation of variables again,

$$\begin{aligned}
 \text{IE11} &= \frac{-m\pi}{2 \sin \alpha} \left[ C_s \int_{W_{si}}^{W_{sf}} \frac{\sin(W_s)}{W_s} dW_s - S_s \int_{W_{si}}^{W_{sf}} \frac{\cos(W_s)}{W_s} dW_s \right] \\
 &\quad \frac{-m\pi}{2 \sin \alpha} \left[ C_d \int_{W_{di}}^{W_{df}} \frac{\sin(W_d)}{W_d} dW_d - S_d \int_{W_{di}}^{W_{df}} \frac{\cos(W_d)}{W_d} dW_d \right]
 \end{aligned}$$

Now subroutine SICI is used. If  $m=k \neq 0$ , the terms involving  $C_d$  and  $S_d$  reduce to zero.

$$\text{IREE11} = -m^2 \pi^2 \int_0^1 \cos m\pi e \cos k\pi e de \quad (\text{F-4})$$

$$= -m^2 \pi^2 \text{IR11}$$

$$\text{IR121} = \text{IR111} \quad (\text{F-5})$$

$$\text{IE21} = \text{IE11} \quad (\text{F-6})$$

$$\text{IR131} = \text{IR111} \quad (\text{F-7})$$

$$\text{IE31} = \text{IE11} \quad (\text{F-8})$$

$$\text{IE12} = \text{IE11} \quad (\text{F-9})$$

$$\text{IR122} = \text{IR111} \quad (\text{F-10})$$

$$\text{IR22} = \text{IR11} \quad (\text{F-11})$$

$$\text{IE22} = \text{IE11} \quad (\text{F-12})$$

$$\text{IREE22} = \text{IREE11} \quad (\text{F-13})$$

$$\text{IR132} = \text{IR111} \quad (\text{F-14})$$

$$\text{I42} = \text{IR11} \quad (\text{F-15})$$

$$\text{IE13} = \text{IE11} \quad (\text{F-16})$$

$$\text{IR133} = \text{IR111} \quad (\text{F-17})$$

$$\text{IR33} = \text{IR11} \quad (\text{F-18})$$

$$\text{IE33} = \text{IE11} \quad (\text{F-19})$$

$$\text{IREE33} = \text{IREE11} \quad (\text{F-20})$$

$$\text{I43} = \text{IR11} \quad (\text{F-21})$$

$$\text{I53} = \int_0^1 R^{-1} \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (\text{F-22})$$

If  $\sin \alpha = 0$ ,

$$\text{I53} = \begin{cases} \bar{R}_0^{-1} & ; m=k=0 \\ \frac{1}{2}\bar{R}_0^{-1} & ; m=k \neq 0 \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , and  $m=k=0$ , then  $\text{I53} = \left\{ \ln[(\bar{R}_0 + \sin \alpha)/\bar{R}_0] \right\} / \sin \alpha$ .

For the general case,  $\sin \alpha \neq 0$ ,  $m \neq k$ , the transformation of variables procedure is used to obtain

$$\begin{aligned} \text{I53} = & \frac{1}{2 \sin \alpha} \left[ C_d \int_{W_{di}}^{W_{df}} \frac{\cos(W_d)}{W_d} dW_d + S_d \int_{W_{di}}^{W_{df}} \frac{\sin(W_d)}{W_d} dW_d \right] \\ & + \frac{1}{2 \sin \alpha} \left[ C_s \int_{W_{si}}^{W_{sf}} \frac{\cos(W_s)}{W_s} dW_s + S_s \int_{W_{si}}^{W_{sf}} \frac{\sin(W_s)}{W_s} dW_s \right] \end{aligned}$$

Now SICI is used to complete the solution. If  $m=k \neq 0$ , the  $C_d$  and  $S_d$  terms are replaced by  $\left\{ \ln[(\bar{R}_0 + \sin \alpha)/\bar{R}_0] \right\} / (2 \sin \alpha)$ .

$$\text{IRE53} = -m\pi \int_0^1 \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (\text{F-23})$$

$$= \begin{cases} \frac{-2m^2}{(m^2 - k^2)} & ; m+k = \text{odd}, m \neq 0 \\ 0 & ; \text{otherwise} \end{cases}$$

$$\text{IR144} = \text{IR11} \quad (\text{F-24})$$

$$IR44 = \int_0^1 R^2 \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (F-25)$$

If  $\sin \alpha = 0$ ,

$$IR44 = \begin{cases} \bar{R}_0^2 & ; m=k=0 \\ \frac{1}{2}\bar{R}_0^2 & ; m=k \neq 0 \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , but  $m=k=0$ ,

$$IR44 = \int_0^1 R^2 \, d\epsilon = \frac{1}{3 \sin \alpha} [(\bar{R}_0 + \sin \alpha)^3 - \bar{R}_0^3].$$

If  $\sin \alpha \neq 0$ , and  $m \neq k$ , Equation (F-25) is changed to

$$IR44 = \frac{1}{2} \int_0^1 R^2 [\cos (m+k)\pi\epsilon + \cos (m-k)\pi\epsilon] \, d\epsilon$$

and the following relation is used

$$\int f(x) \cos a x \, dx = \frac{\sin a x}{a} \left[ f - \frac{f_{,xx}}{2} + \dots \right] + \frac{\cos a x}{a} \left[ f_{,x} - \frac{f_{,xxx}}{3} + \dots \right].$$

Then,

$$IR44 = \frac{\sin \alpha}{\pi^2 (m+k)^2} \left\{ \cos (m+k)\pi [\bar{R}_0 + \sin \alpha] - \bar{R}_0 \right\} \\ + \frac{\sin \alpha}{\pi^2 (m-k)^2} \left\{ \cos (m-k)\pi [\bar{R}_0 + \sin \alpha] - \bar{R}_0 \right\}$$

If  $\sin \alpha \neq 0$ , but  $m=k \neq 0$ , the  $(m-k)$  term in the above equation is replaced by  $[(\bar{R}_0 + \sin \alpha)^3 - \bar{R}_0^3]/6 \sin \alpha$ .

$$IE44 = -m\pi \int_0^1 R \sin m\pi\epsilon \cos k\pi\epsilon \, d\epsilon$$

$$+ 2 \sin \alpha \int_0^1 \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon \quad (\text{F-26})$$

If  $\sin \alpha = 0$ ,

$$\text{IE44} = \begin{cases} \frac{-2m^2 \bar{R}_0}{(m^2 - k^2)} & ; m+k = \text{odd}, m \neq 0 \\ 0 & ; \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ , Equation (F-26) transforms to

$$\begin{aligned} \text{IE44} &= -\frac{m\pi}{2} \int_0^1 R [\sin (m+k)\pi\epsilon + \sin (m-k)\pi\epsilon] d\epsilon \\ &+ 2 \sin \alpha \int_0^1 \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon, \end{aligned}$$

and making use of

$$\begin{aligned} \int f(x) \sin a x dx &= \frac{\sin a x}{a} \left[ f, x - \frac{f, xxx}{a^3} + \dots \right] \\ &- \frac{\cos a x}{a} \left[ f - \frac{f, xx}{2} + \dots \right]. \end{aligned}$$

Then,

$$\begin{aligned} \text{IE44} &= \frac{m\pi}{2} \left[ \frac{\cos (m+k)\pi}{\pi(m+k)} (\bar{R}_0 + \sin \alpha) - \frac{\bar{R}_0}{\pi(m+k)} + \frac{\cos (m-k)\pi}{\pi(m-k)} (\bar{R}_0 \right. \\ &\quad \left. + \sin \alpha) - \frac{\bar{R}_0}{\pi(m-k)} \right] \\ &+ 2 \sin \alpha \begin{cases} 1 & ; m=k=0 \\ \frac{1}{2} & ; m=k \neq 0 \\ 0 & ; \text{otherwise} \end{cases} \end{aligned}$$

The terms involving  $(m-k)$  reduce to zero when  $m=k$ .

$$\text{IREE44} = -m^2 \pi^2 \int_0^1 R^2 \cos m\pi\epsilon \cos k\pi\epsilon \, d\epsilon$$

$$- 2 m \pi \sin \alpha \int_0^1 R \sin m \pi \epsilon \cos k \pi \epsilon \, d\epsilon \quad (\text{F-27})$$

If  $m=k=0$ ,  $\text{IREE44} = 0$ .

If  $\sin \alpha = 0$ ,

$$\text{IREE44} = \begin{cases} \frac{-m^2 \pi^2 \bar{R}_o^2}{2} & ; \quad m=k \neq 0 \\ 0 & ; \quad \text{otherwise} \end{cases}$$

If  $\sin \alpha \neq 0$ ,

$$\begin{aligned} \text{IREE44} &= -m^2 \pi^2 \text{IR44} - m \pi \sin \alpha \int_0^1 R [\sin (m+k) \pi \epsilon + \sin (m-k) \pi \epsilon] \, d\epsilon \\ &= -m^2 \pi^2 \text{IR44} + m \pi \sin \alpha \left[ \frac{\cos (m+k) \pi}{\pi (m+k)} (\bar{R}_o + \sin \alpha) - \frac{\bar{R}_o}{\pi (m+k)} \right. \\ &\quad \left. + \frac{\cos (m-k) \pi}{\pi (m-k)} (\bar{R}_o + \sin \alpha) - \frac{\bar{R}_o}{\pi (m-k)} \right] \end{aligned}$$

If  $m=k$ , the  $(m-k)$  terms become zero.

$$\text{IR154} = \text{I53} \quad (\text{F-28})$$

$$\text{IE54} = \text{IRE53} \quad (\text{F-29})$$

$$\text{IRE35} = \text{IRE53} \quad (\text{F-30})$$

$$\begin{aligned} \text{IE45} &= -m \pi \int_0^1 \sin m \pi \epsilon \cos k \pi \epsilon \, d\epsilon \\ &\quad + \sin \alpha \int_0^1 R^{-1} \cos m \pi \epsilon \cos k \pi \epsilon \, d\epsilon \quad (\text{F-31}) \end{aligned}$$

$$= \text{IRE53} + (\sin \alpha) \text{I53}$$

$$\text{IR155} = \text{IR111} \quad (\text{F-32})$$

$$\text{IR55} = \text{IR11} \quad (\text{F-33})$$

$$\text{IE55} = \text{IE11} \quad (\text{F-34})$$

$$\text{IREE55} = \text{IREE11} \quad (\text{F-35})$$

## APPENDIX G

### COMPUTER PROGRAM DOCUMENTATION

The program is written in E-level Fortran IV, and was run using an IBM System 360, Model 40 (with 128K bytes of core storage) under Release 14 of the OS System. The program calls for three scratch tapes.

A small control program serves as the main program, while all the operations are done in subprograms. This organization facilitates the use of a rather extensive overlay structure, whereby only parts of the procedure are in core storage at one time.

The flow of the program is summarized as follows: The control program calls subroutine PART1. PART1 generates the stiffness and inertia matrices by reading all input data, calculating the constants defined by Equations (2-8), calling the function subprograms for the various integrations, and arranging the submatrices to form the stiffness and inertia matrices. Each integral value is stored so that it need not be recalculated. If desired the various constants, the integral values, and the generated matrices may be printed. Normally, the input data are written on a scratch tape and the matrices are written on another scratch tape. Subroutine CHECK, which is called optionally by PART1, is used to test the flow of PART1.

Subroutine PART2 is called from the control program and

performs a matrix inversion and a matrix multiplication by calling subroutines DARRAY, DMINV, FARRAY, and MULTM1. PART2 first reads the matrices from the scratch tape. DARRAY simply transforms a double-dimension array into a single-dimension array (or vice-versa). DMINV performs a matrix inversion of the stiffness matrix. FARRAY is identical to DARRAY but because of the overlay procedure, must be included. FARRAY is used to transform the inverted stiffness matrix back to double-dimension. Subroutine MULTM1 is used to multiply the inverted stiffness matrix times the inertia matrix. The resultant matrix is written on the third scratch tape.

The control program next calls subroutine PART2 which reads the resultant matrix from the third scratch tape. It is used to calculate the frequencies and mode shapes. The input data are read from the first scratch tape and printed by the calling of subroutine WRITE1.

The eigenvalues and eigenvectors are found by the direct and inverse power methods in subroutine MATSUB. Then subroutine WRITE2 is called by PART3 to print the frequencies and mode shapes.

A thorough explanation of the subroutines ARRAY (DARRAY and FARRAY) and MINV (DMINV), as well as SICI and QTFE, may be found in Reference [36]. An explanation of subroutine MATSUB, in its original form, is given in Reference [38]. A diagram of the overlay structure for the clamped-clamped boundary condition is shown in Figure G.1. For the free-free boundary condition, the only change is to move the function subprograms IR111, IE11, and IREE11 into the control section with PART1. For the freely supported condition, all the function

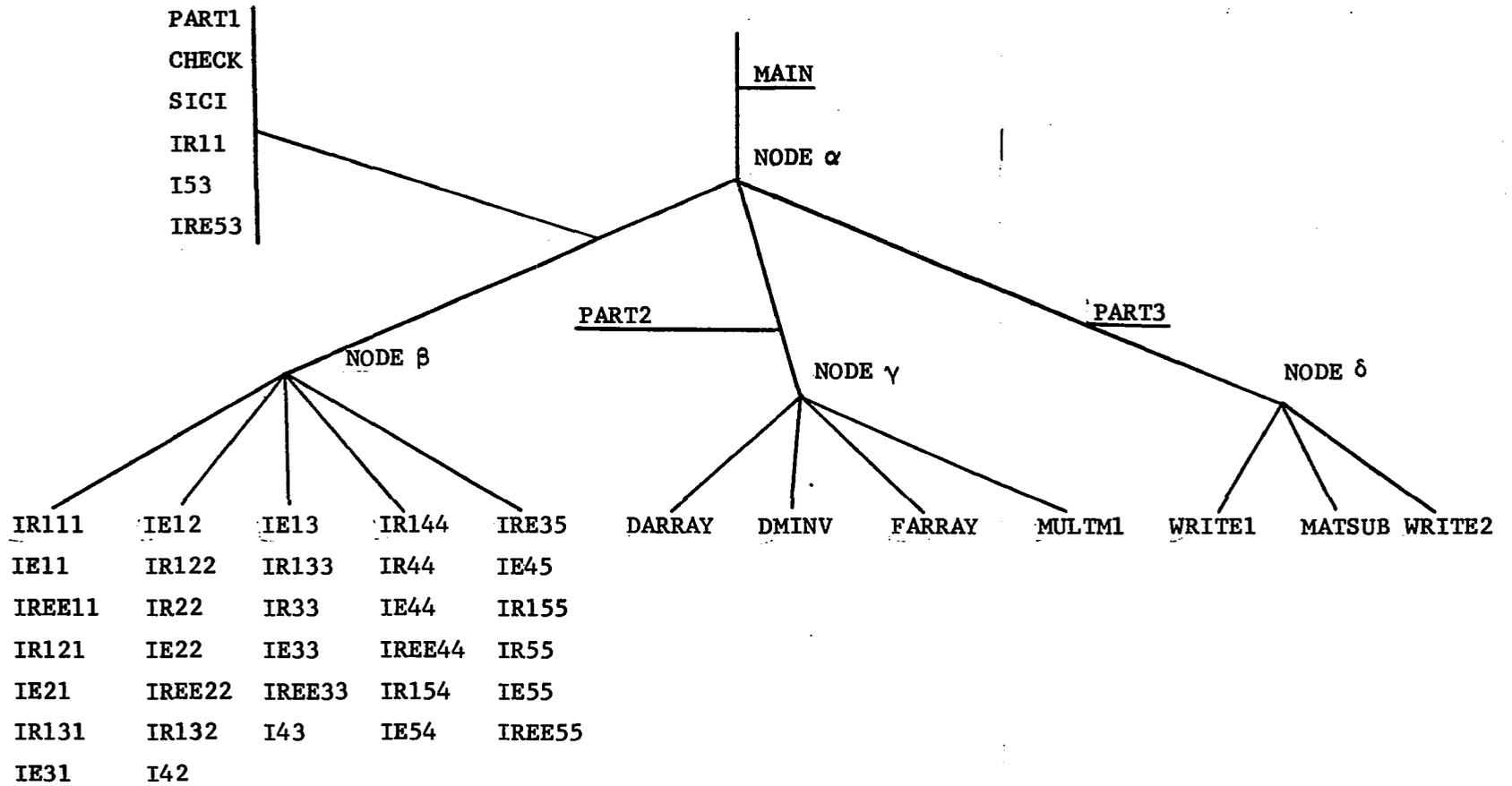


Figure G.1 Overlay Structure

subprograms are moved into the appropriate control section under node beta and subroutine QTFE is moved into the control section with PART1.

The input data deck is set up as follows:

1. Title card identifying case being run.
2. Boundary condition being used.
3. Shell geometry and Poisson's ratios.
4. Modulus data.
5. Densities and shear coefficients.
6. Control card defining flow of problem.
7. Control card for eigenvalue solution.
8. Starting indices for assumed modes.
9. Number of terms in assumed modes.
10. Values of n to be run.

Cases may be stacked as long as the boundary condition remains the same.

The formats for the above cards are:

1. (10A8) NAME (I), I=1,10

All eighty columns are available to assign a descriptive title to the problem being run. The title should be centered on the card in order to be centered on the printed output.

2. (3A8) BCOND (I), I=1,3

The first twenty-four columns are used to write the boundary condition. The punches should start in column 1.

3. (7F10.0) ANG, RO, XL, T, H, MUX, MUT

ANG = Shell semi-vertex angle,  $\alpha$ . (degrees). For a cylinder,

$$\alpha = 0.$$

- RO = Shell small-end radius,  $R_o$ . (inches)
- XL = Shell slant length, L. (inches)
- T = Facing half-thickness, t. (inches) For a homogeneous shell,  $T = \frac{1}{2}$  of shell thickness.
- H = Core half-thickness, h. (inches) For a homogeneous shell,  $H = 0$ .
- MUX = Major Poisson's ratio,  $\nu'_{\theta x}$ . (Dimensionless)
- MUT = Minor Poisson's ratio,  $\nu'_{x\theta}$ . (Dimensionless)
4. (7D10.6) EX, ET, GZXF, GTZF, GZXC, GTZC
- EX = Facing elastic modulus in x-direction,  $E'_x$ . (psi)
- ET = Facing elastic modulus in  $\theta$ -direction,  $E'_\theta$ . (psi)
- GZXF = Facing shear modulus in z-x plane,  $G'_{zx}$ . (psi)
- GTZF = Facing shear modulus in  $\theta$ -z plane,  $G'_{\theta z}$ . (psi)
- GXTF = Facing shear modulus in x- $\theta$  plane,  $G'_{x\theta}$ . (psi)
- GZXC = Core shear modulus in z-x plane,  $G_{zx}$ . (psi)
- GTZC = Core shear modulus in  $\theta$ -z plane,  $G_{\theta z}$ . (psi)
5. (D10.6, 2F10.0, D10.6, 2F10.0) RF, KXF, KTF, RC, KXC, KTC
- RF = Facing density,  $\rho'$ . (lb-sec.<sup>2</sup>/in.<sup>4</sup>)
- KXF = Facing shear coefficient in z-x plane,  $K'_x$ . (Dimensionless)
- KTF = Facing shear coefficient in  $\theta$ -z plane,  $K'_\theta$ .
- RC = Core density,  $\rho$ . (lb-sec.<sup>2</sup>/in.<sup>4</sup>)
- KXC = Core shear coefficient in z-x plane,  $K_x$ . (Dimensionless)
- KTC = Core shear coefficient in  $\theta$ -z plane,  $K_\theta$ . (Dimensionless)
6. (7I10) NWRITE, NCHECK, NTAPE, NCASE, NTERM, NTAGBC, NTEST
- NWRITE = Control indicating whether or not the stiffness and inertia matrices are printed out. Equals 1 for print and equals 2 for no print.

- NCHECK = Control indicating whether or not the flow of PART1 needs to be monitored by printing out indicators with subroutine CHECK. Equals 1 for check and equals 2 for no check. Normally, equals 2.
- NTAPE = Control indicating whether or not the stiffness and inertia matrices are written on the scratch tape. Equals 1 for tape and equals 2 for no tape. Normally, equals 1.
- NCASE = Control indicating whether or not another complete case is stacked behind present case. Equals 2 for yes and equals 1 for no.
- NTERM = Control indicating whether or not the present case is to be repeated after changing the starting index or number of terms in assumed series. Equals 2 for yes and equals 1 for no. If yes, only card number 6 and subsequent cards are included in next data stack.
- NTAGBC = Control indicating whether the stiffness matrix or the inertia matrix is inverted in PART2. Equals 2 for inverting inertia matrix and equals 1 for inverting stiffness matrix. Normally, the stiffness matrix is inverted so that the eigenvalues found are the reciprocals of the square of the frequencies. Since the eigenvalue program iterates to the larger eigenvalues first, this allows calculation of the desired number of lowest frequencies. However, if problems arise in trying to invert the stiffness matrix, the inertia matrix may always be inverted. In the case when the inertia matrix is inverted, the eigenvalue is the square of the frequency, and all of the frequencies must be found to obtain the lowest.
- NTEST = Control indicating whether or not the ETA's [Equations (A-20)], the C's [Equations (2-8)], and the values of the integrals are printed. Equals 1 for yes and equals 2 for no.
7. (5I4, 2D10.6, I10) IEG, IVEC, IDET, MIT, MITS, ALRS, GBR, IQUIT
- IEG = Control indicating whether or not every iteration for the eigenvalue is printed. Equals 1 for yes and equals 0 for no. Normally, equals 0.
- IVEC = Control indicating whether or not eigenvectors are desired. Equals 1 for yes and equals 0 for no. Normally, equals 1.
- IDET = 1
- MIT = Maximum number of iterations for direct power method.

MITS = Maximum number of iterations for inverse power method.  
 ALR = Initial eigenvalue guess. Normally, ALR = 1.0  
 GBR = Increment added to current eigenvalue if inverse power method will not converge on first try. Normally, GBR = 0.0.  
 IQUIT = Number of eigenvalues desired. Must equal order of matrix if NTAGBC = 2.

8. (7I10) MI(I), I = 1, 7

Starting index for assumed mode series. Each equals 1 for clamped-clamped and freely supported and each equals 0 for free-free.

9. (7I10) NT(I), I = 1, 7

Number of terms for each assumed function. Maximum is six.

10. (2I10) N, NMAX

N = Circumferential wave number.

NMAX = Control indicating whether or not other N-values follow. If NMAX is any number greater than N, another card of the form (10.) follows, and the procedure is repeated (without having to recalculate the integrals) after changing only N. If NMAX = N, the program does a normal stop after the eigenvalues are printed.

On the following pages, the stacking of a problem is shown, along with the necessary control cards, and including a set of typical input data. It is noted that a card with a /\* in columns 1 and 2 must be included immediately following the statement "ALL SOURCE PROGRAM DECKS", immediately following the statement "INSERT WRITE2", and immediately following the last card of input data.

JOB STACKING

```
*****
JOB CARD
*****
//FORT EXEC PGM=IEJFAA0
//SYSPRINT DD SYSOUT=A,DCB=BLKSIZE=121
//SYSPUNCH DD UNIT=SYSP,DCB=BLKSIZE=80
//SYSUT1 DD UNIT=SYSSQ,SE=SYSPUNCH,SPACE=(904,(30,20))
//SYSUT2 DD UNIT=SYSSQ,SEP=SYSUT1,SPACE=(904,(30,20))
//SYSLIN DD UNIT=SYSSQ,SEP=SYSPUNCH,DSNAME=&LOADSET,DISP=(MOD,PASS), X
// SPACE=(80,(400,400),RLSE)
//SYSIN DD *
*****
ALL SOURCE PROGRAM DECKS
*****
//LKED EXEC PGM=IEWL,PARM=(OVLY,LET), X
// REGION=96K
//SYSPRINT DD SYSOUT=A,DCB=BLKSIZE=121
//SYSLIB DD DSNAME=SYS1.FORTLIB,DISP=SHR
//SYSLMOD DD DSNAME=&GOSET(MAIN),DISP=(NEW,PASS),UNIT=SYSDA, X
// SPACE=(CYL,(70,20,1),RLSE,MXIG)
//SYSUT1 DD UNIT=(SYSDA,SEP=(SYSLMOD,SYSLIB)), X
// SPACE=(CYL,(20,20),,MXIG)
//SYSLIN DD DSNAME=&LOADSET,DISP=(CLO,DELETE),DCB=BLKSIZE=80
// DD *
INSERT MAIN
OVERLAY ALPHA
INSERT PART1,CHECK,SICI,QTFE
OVERLAY BETA
INSERT IP111,IR11,IE11,IREE11,IR121,IE21,IR131,IE31
OVERLAY BETA
INSERT IE12,IR122,IR22,IE22,IRFE22,IR132,I42
OVERLAY BETA
INSERT IE13,IR133,IR33,IE33,IREE33,I43,I53,IRE53
```

```

OVERLAY BETA
INSERT IR144,IR44,IE44,IREE44,IR154,IE54
OVERLAY BETA
INSERT IRE35,IE45,IR155,IR55,IE55,IREE55
OVERLAY ALPHA
INSERT PART2
OVERLAY GAMMA
INSERT DARRAY
OVERLAY GAMMA
INSERT DMINV
OVERLAY GAMMA
INSERT FARRAY
OVERLAY GAMMA
INSERT MULTM1
OVERLAY ALPHA
INSERT PART3
OVERLAY DELTA
INSERT WRITE1
OVERLAY DELTA
INSERT MATSUB
OVERLAY DELTA
INSERT WRITE2
//GO EXEC PGM=*.LKED.SYSLMOD
//FT03F001 DD SYSOUT=A
//FT06F001 DD SYSOUT=A
//FT02F001 DD UNIT=SYSCP
//GO.FT07F001 DD UNIT=180,LABEL=(,NL),VOLUME=SER=AAA,DSNAME=BBB,      X
//          DCB=BUFNO=1
//GO.FT08F001 DD UNIT=181,LABEL=(,NL),VOLUME=SER=AAB,DSNAME=BBC,      X
//          DCB=BUFNO=1
//GO.FT09F001 DD UNIT=182,LABEL=(,NL),VOLUME=SER=AAC,DSNAME=BBB,      X
//          DCB=BUFNO=1
//FT01F001 DD *
*****
          INPUT DATA ( TYPICAL )
          1968 O.U. SANDWICH CONE - FIBERGLASS FACINGS AND ALUMINUM HONEYCOMB CORE

```

FREE - FREE

5.07	22.45	72.5	.0105	.15	.2	.2
3.64D+06	3.64D+06	1.00D+06	1.00D+06	1.00D+06	.320D+05	.183D+05
.2652 D-03	1.	1.	.3368 D-05	1.	1.	
	2	2	1	1	1	2
0	1	1 50 10	1.00		4	
	0	0	0	0	0	0
	6	6	6	6	6	6
	2	2				
*****						

---

APPENDIX H

COMPUTER PROGRAM LISTING

C CONTROL PROGRAM FOR ANALYSIS OF SYMMETRIC AND UNSYMMETRIC FREE  
C VIBRATIONS OF CONICAL OR CYLINDRICAL SANDWICH SHELLS, USING  
C GALERKIN'S METHOD.  
C

CALL PART1  
CALL PART2  
CALL PART3  
STOP  
END

C SUBROUTINE PART1  
C CALCULATION OF STIFFNESS AND INERTIA MATRICES  
C

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

DOUBLE PRECISION AS(42,42),AI(42,42)

DOUBLE PRECISION NAME(10),BCOND(3)

DOUBLE PRECISION ANG,RO,XL,T, MUX,MUT,EX,ET,GZXF,GTZF,GXTF,GZXC,  
1 GTZC,RF,KXF,KTF,RC,KXC,KTC,RAD,AA,EXB,ETB,MB,JC,JF,CA,SA2,CA2,  
2 EXL,EXL3,HH

DOUBLE PRECISION ALRS,GBR

DOUBLE PRECISION IR111,IR11,IE11,IREE11,IR121,IE21,IR131,IE31,  
1IE12,IR122,IR22,IE22,IREE22,IR132,I42,IE13,IR133,IR33,IE33,IREE33,  
2I43,I53,IRE53,IR144,IR44,IE44,IREE44,IR154,IE54,IRE35,IE45,IR155,  
3IR55,IE55,IREE55

REAL N1,N2,N3,N4,N5,N6,N7,N8,N9,N10,N11,N12,N13,N14,N15,N16

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,

```
3FE55,FREE55,H,XMU,NT(7)
  DIMENSION KI(7),MI(7)
  EQUIVALENCE (KI(1),MI(1))
  ITAPE=1
```

C  
C  
C

READ CASE IDENTIFICATION, BOUNDARY CONDITIONS, & SHELL PROPERTIES

```
1 READ(1,2)(NAME(I),I=1,10)
2 FORMAT(10A8)
  READ(1,3)(BCOND(I),I=1,3)
3 FORMAT(3A8)
  READ(1,4)ANG,RD,XL,T,H,MUX,MUT
4 FORMAT(7F10.0)
  READ(1,5)EX,ET,GZXF,GTZF,GXTF,GZXC,GTZC
5 FORMAT(7D10.6)
  READ(1,6)RF,KXF,KTF,RC,KXC,KTC
6 FORMAT(D10.6,2F10.0,D10.6,2F10.0)
```

C  
C  
C

CALCULATE VARIOUS CONSTANTS

```
PI=3.141592653589793
RAD=57.29577951308232
AA=H+T
EXB=EX/(1.DO-MUT*MUX)
ETB=ET/(1.DO-MUT*MUX)
MB=2.DO*(RC*H+2.DO*RF*T)
JC=2.DO*RC*H*H*H/3.DO
JF=2.DO*RF*T*(T*T/3.DO+AA*AA)
N1=2.*T*EXB
N2=2.*T*ETB
N3=2.*T*(MUX*EXB+MUT*ETB)
N4=2.*T*KXF*GZXF
N5=2.*T*KTF*GTZF
N6=4./3.*T*T*N1
N7=4./3.*T*T*N2
N8=4./3.*T*T*N3
```



```

C      = 1, THE STIFFNESS MATRIX IS INVERTED AND ONLY THE DESIRED
C      NUMBER OF FREQUENCIES MAY BE CALCULATED, STARTING FROM THE
C      MINIMUM.
C
C      IF NTEST = 2, THE N'S, THE C'S, AND THE INTEGRALS ARE NOT
C      PRINTED. OTHERWISE, NTEST = 1 .
C
C      7 READ(1,8)NWRITE,NCHECK,NTAPE,NCASE,NTERM,NTAGBC,NTEST
C      8 FORMAT(7I10)
C      READ (1,11) IEG,IVEC,IDET,MIT,MITS,ALRS,GBR,IQUIT
C      11 FORMAT (5I4,2D10.6,I10)
C      GOTO(10001,20001),NCHECK
C
C      10001 CALL CHECK ( 1)
C      20001 CONTINUE
C
C      READ STARTING INDEX FOR EACH SERIES , & NUMBER OF TERMS IN EACH
C
C      NOTE: THE ASSUMED SERIES SOLUTIONS FOR PSI X & PSI X PRIME MUST
C      BE IDENTICAL. THE SAME IS TRUE FOR PSI THETA & PSI THETA PRIME.
C
C      9 READ(1,8)(MI(I),I=1,7)
C      READ(1,8)(NT(I),I=1,7)
C
C      GOTO(10002,20002),NCHECK
C      10002 CALL CHECK ( 2)
C      20002 CONTINUE
C      NTS2=NT(1)+NT(2)
C      NTS3=NTS2+NT(3)
C      NTS4=NTS3+NT(4)
C      NTS5=NTS4+NT(5)
C      NTS6=NTS5+NT(6)
C      NTS7=NTS6+NT(7)
C      ICALC=1
C
C      READ & WRITE N
C

```



C45=2.DO\*N\*SA\*(N7+N14)/EXL3  
 C451=-N\*(N8+2.DO\*N14)/EXL3  
 C46=(N\*N\*N11+2.DO\*N5\*H\*T\*CA2+N13\*SA2)/EXL3  
 C462=-N13/EXL3  
 C461=SA\*C462  
 C47=N\*SA\*(N11+N13)/EXL3  
 C471=-N\*(N10+N13)/EXL3  
 C55=2.DO\*(N7\*SA2+N\*N\*N14)/EXL3  
 C550=2.DO\*N4/EXL  
 C552=-2.DO\*N6/EXL3  
 C551=SA\*C552  
 C56=N\*SA\*(N11+N13)/EXL3  
 C561=-N\*(N10+N13)/EXL3  
 C57=(N11\*SA2+N\*N\*N13)/EXL3  
 C572=-N9/EXL3  
 C571=SA\*C572  
 C66=2.DO\*H\*H\*(N5\*CA2+N12\*SA2+N\*N\*N2)/EXL3  
 C660=2.DO\*N16/EXL  
 CJC=4.DO\*PI\*PI\*JC/EXL  
 C662=-2.DO\*HH\*N12/EXL3  
 C661=SA\*C662  
 C67=+2.DO\*N\*HH\*SA\*(N2+N12)/EXL3  
 C671=-N\*HH\*(N3+2.DO\*N12)/EXL3  
 C77=2.DO\*HH\*(N2\*SA2+N\*N\*N12)/EXL3  
 C770=2.DO\*N15/EXL  
 C772=-2.DO\*N1\*HH/EXL3  
 C771=SA\*C772  
 GOTO(10003,20003),NCHECK

10003 CALL CHECK ( 3)  
 20003 CONTINUE

C  
 C INITIALIZE MATRICES

C  
 DO 600 I=1,NTS7  
 DO 600 J=1,NTS7  
 AS(I,J)=0.DO

```

600 AI(I,J)=0.DO
C
C   CALCULATE STIFFNESS AND INERTIA MATRICES
C
C   FIRST SET OF ROWS
C
C
      KKF=NT(1)
      DO 1000 KK=1,KKF
      II=KK
      IF(KI(1))1002,1001,1002
1001 K=KK-1
      GO TO 1003
1002 K=KK
1003 CONTINUE
C
C   SUBMATRIX U - U
C
      MMF=NT(1)
      DO 1100 MM=1,MMF
      JJ=MM
      IF(MI(1))1012,1011,1012
1011 M=MM-1
      GO TO 1013
1012 M=MM
1013 GO TO (1014,1015),ICALC
1014 FR11(KK,MM)=IR11(K,M)
      FR11(KK,MM)=IR11(K,M)
      FE11(KK,MM)=IE11(K,M)
      FREE11(KK,MM)=IREE11(K,M)
1015 AI(II,JJ)=CM *FR11(KK,MM)
      AS(II,JJ)=C11*FR11(KK,MM)+C111*FE11(KK,MM)+C112*FREE11(KK,MM)
1100 CONTINUE
      GOTO(1004,20004),NCHECK
10004 CALL CHECK ( 4)

```

20004 CONTINUE

C  
C  
C

SUBMATRIX U - V

```
MMF=NT(2)
DO 1200 MM=1,MMF
  JJ=MM+NT(1)
  IF(N)1102,1200,1102
1102 IF(MI(2))1104,1103,1104
1103 M=MM-1
  GO TO 1105
1104 M=MM
1105 GO TO (1106,1107),ICALC
1106 FR121(KK,MM)=IR121(K,M)
  FE21(KK,MM)=IE21(K,M)
1107 AS(II,JJ)=C12*FR121(KK,MM)+C121*FE21(KK,MM)
1200 CONTINUE
  GOTO(10005,20005),NCHECK
10005 CALL CHECK ( 5)
20005 CONTINUE
```

C  
C  
C

SUBMATRIX U - W

```
MMF=NT(3)
DO 1300 MM=1,MMF
  JJ=MM+NTS2
  IF(MI(3))1202,1201,1202
1201 M=MM-1
  GO TO 1203
1202 M=MM
1203 GO TO (1204,1205),ICALC
1204 FR131(KK,MM)=IR131(K,M)
  FE31(KK,MM)=IE31(K,M)
1205 AS(II,JJ)=C13*FR131(KK,MM)+C131*FE31(KK,MM)
1300 CONTINUE
  GOTO(10006,20006),NCHECK
```

```

10006 CALL CHECK ( 6)
20006 CONTINUE
  1000 CONTINUE
C
C   SECOND SET OF ROWS
C
      KKF=NT(2)
      DO 2000 KK=1,KKF
      II=KK+NT(1)
      IF(KI(2))2002,2001,2002
2001 K=KK-1
      GO TO 2003
2002 K=KK
2003 CONTINUE
C
C   SUBMATRIX V - U
C
      MMF=NT(1)
      DO 2100 MM=1,MMF
      JJ=MM
      IF(N)2012,2100,2012
2012 IF(MI(1))2014,2013,2014
2013 M=MM-1
      GO TO 2015
2014 M=MM
2015 GO TO (2016,2017),ICALC
2016 FE12(KK,MM)=IE12(K,M)
2017 AS(II,JJ)=C12*FR121(MM,KK)-C121*FE12(KK,MM)
2100 CONTINUE
      GOTO(10007,20007),NCHECK
10007 CALL CHECK ( 7)
20007 CONTINUE
C
C   SUBMATRIX V - V
C
      MMF=NT(2)

```

```

      DO 2200 MM=1,MMF
      JJ=MM+NT(1)
      IF(MI(2))2102,2101,2102
2101 M=MM-1
      GO TO 2103
2102 M=MM
2103 GO TO (2104,2105),ICALC
2104 FR122(KK,MM)=IR122(K,M)
      FR22(KK,MM)=IR22(K,M)
      FE22(KK,MM)=IE22(K,M)
      FREE22(KK,MM)=IREE22(K,M)
2105 AI(II,JJ)=CM *FR22(KK,MM)
      AS(II,JJ)=C22*FR122(KK,MM)+C221*FE22(KK,MM)+C222*FREE22(KK,MM)
2200 CONTINUE
      GOTO(10008,20008),NCHECK
10008 CALL CHECK ( 8)
20008 CONTINUE
C
      SUBMATRIX V - W
C
      MMF=NT(3)
      DO 2300 MM=1,MMF
      JJ=MM+NTS2
      IF(N)2202,2300,2202
2202 IF(MI(3))2204,2203,2204
2203 M=MM-1
      GO TO 2205
2204 M=MM
2205 GO TO (2206,2207),ICALC
2206 FR132(KK,MM)=IR132(K,M)
2207 AS(II,JJ)=C23*FR132(KK,MM)
2300 CONTINUE
      GOTO(10009,20009),NCHECK
10009 CALL CHECK ( 9)
20009 CONTINUE
C

```

C SUBMATRIX V - PSI THETA PRIME  
C

MMF=NT(4)  
DO 2500 MM=1,MMF  
JJ=MM+NTS3  
IF(MI(4))2402,2401,2402  
2401 M=MM-1  
GO TO 2403  
2402 M=MM  
2403 GO TO (2404,2405),ICALC  
2404 F42(KK,MM)=I42(K,M)  
2405 AS(II,JJ)=C24\*F42(KK,MM)  
2500 CONTINUE

C  
C SUBMATRIX V - PSI THETA  
C

MMF=NT(6)  
DO 2400 MM=1,MMF  
JJ=MM+NTS5  
IF(H)2302,2400,2302  
2302 AS(II,JJ)=C26\*F42(KK,MM)  
2400 CONTINUE  
GOTO(10010,20010),NCHECK  
10010 CALL CHECK (10)  
20010 CONTINUE  
2000 CONTINUE

C  
C THIRD SET OF ROWS  
C

KKF=NT(3)  
DO 3000 KK=1,KKF  
II=KK+NTS2  
IF(KI(3))3002,3001,3002  
3001 K=KK-1  
GO TO 3003  
3002 K=KK

3003 CONTINUE

C  
C  
C

SUBMATRIX W - U

MMF=NT(1)

DO 3100 MM=1,MMF

JJ=MM

IF(MI(1))3012,3011,3012

3011 M=MM-1

GO TO 3013

3012 M=MM

3013 GO TO (3014,3015),ICALC

3014 FE13(KK,MM)=IE13(K,M)

3015 AS(II,JJ)=C13\*FR131(MM,KK)-C131\*FE13(KK,MM)

3100 CONTINUE

GOTO(10011,20011),NCHECK

10011 CALL CHECK (11)

20011 CONTINUE

C  
C  
C

SUBMATRIX W - V

MMF=NT(2)

DO 3200 MM=1,MMF

JJ=MM+NT(1)

IF(N)3102,3200,3102

3102 AS(II,JJ)=C23\*FR132(MM,KK)

3200 CONTINUE

GOTO(10012,20012),NCHECK

10012 CALL CHECK (12)

20012 CONTINUE

C  
C  
C

SUBMATRIX W - W

MMF=NT(3)

DO 3300 MM=1,MMF

JJ=MM+NTS2

```

      IF(MI(3))3202,3201,3202
3201 M=MM-1
      GO TO 3203
3202 M=MM
3203 GO TO (3204,3205),ICALC
3204 FR133(KK,MM)=IR133(K,M)
      FR33(KK,MM)=IR33(K,M)
      FE33(KK,MM)=IE33(K,M)
      FREE33(KK,MM)=IREE33(K,M)
3205 AI(II,JJ)=CM *FR33(KK,MM)
      AS(II,JJ)=C33*FR133(KK,MM)+C331*FE33(KK,MM)+C332*FREE33(KK,MM)
3300 CONTINUE
      GOTO(10013,20013),NCHECK
10013 CALL CHECK (13)
20013 CONTINUE

```

```

C
C   SUBMATRIX W - PSI THETA PRIME
C

```

```

      MMF=NT(4)
      DO 3700 MM=1,MMF
      JJ=MM+NTS3
      IF(N)3602,3700,3602
3602 IF(MI(4))3604,3603,3604
3603 M=MM-1
      GO TO 3605
3604 M=MM
3605 GO TO (3606,3607),ICALC
3606 F43(KK,MM)=I43(K,M)
3607 AS(II,JJ)=C34*F43(KK,MM)
3700 CONTINUE

```

```

C
C   SUBMATRIX W - PSI X PRIME
C

```

```

      MMF=NT(5)
      DO 3500 MM=1,MMF
      JJ=MM+NTS4

```

```

      IF(MI(5))3402,3401,3402
3401 M=MM-1
      GO TO 3403
3402 M=MM
3403 GO TO (3404,3405),ICALC
3404 F53(KK,MM)=I53(K,M)
      FRE53(KK,MM)=IRE53(K,M)
3405 AS(II,JJ)=C35*F53(KK,MM)+C351*FRE53(KK,MM)
3500 CONTINUE
      GOTO(10015,20015),NCHECK
10015 CALL CHECK (15)
20015 CONTINUE
C
      SUBMATRIX W - PSI THETA
C
      MMF=NT(6)
      DO 3600 MM=1,MMF
      JJ=MM+NTS5
      IF(N)3502,3600,3502
3502 IF(H)3503,3600,3503
3503 AS(II,JJ)=C36*F43(KK,MM)
3600 CONTINUE
      GOTO(10016,20016),NCHECK
10016 CALL CHECK (16)
20016 CONTINUE
C
      SUBMATRIX W - PSI X
C
      MMF=NT(7)
      DO 3400 MM=1,MMF
      JJ=MM+NTS6
      IF(H)3302,3400,3302
3302 AS(II,JJ)=C37*F53(KK,MM)+C371*FRE53(KK,MM)
3400 CONTINUE
      GOTO(10014,20014),NCHECK
10014 CALL CHECK (14)

```

20014 CONTINUE

3000 CONTINUE

C

C FOURTH SET OF ROWS

C

KKF=NT(4)

DO 7000 KK=1,KKF

II=KK+NTS3

IF(KI(4))7002,7001,7002

7001 K=KK-1

GO TO 7003

7002 K=KK

7003 CONTINUE

C

C SUBMATRIX PSI THETA PRIME - V

C

MMF=NT(2)

DO 7500 MM=1,MMF

JJ=MM+NT(1)

7403 AS(II,JJ)=C24\*F42(MM,KK)

7500 CONTINUE

C

C SUBMATRIX PSI THETA PRIME - W

C

MMF=NT(3)

DO 7600 MM=1,MMF

JJ=MM+NTS2

IF(N)7502,7600,7502

7502 AS(II,JJ)=C34\*F43(MM,KK)

7600 CONTINUE

C

C SUBMATRIX PSI THETA PRIME - PSI THETA PRIME

C

MMF=NT(4)

DO 7400 MM=1,MMF

JJ=MM+NTS3

```

      IF(MI(4))7302,7301,7302
7301 M=MM-1
      GO TO 7303
7302 M=MM
7303 GO TO (7304,7305),ICALC
7304 FR144(KK,MM)=IR144(K,M)
      FR44(KK,MM)=IR44(K,M)
      FE44(KK,MM)=IE44(K,M)
      FREE44(KK,MM)=IREE44(K,M)
7305 AI(II,JJ)=CJF*FR44(KK,MM)
      AS(II,JJ)=C44*FR144(KK,MM)+C441*FE44(KK,MM)+C442*FREE44(KK,MM)
      1 +C440*FR44(KK,MM)
7400 CONTINUE
C
C   SUBMATRIX PSI THETA PRIME - PSI X PRIME
C
      MMF=NT(5)
      DO 7200 MM=1,MMF
      JJ=MM+NTS4
      IF(N)7102,7200,7102
7102 IF(MI(5))7104,7103,7104
7103 M=MM-1
      GO TO 7105
7104 M=MM
7105 GO TO (7106,7107),ICALC
7106 FE54(KK,MM)=IE54(K,M)
      FR154(KK,MM)=IR154(K,M)
7107 AS(II,JJ)=C45*FR154(KK,MM)-C451*FE54(KK,MM)
7200 CONTINUE
C
C   SUBMATRIX PSI THETA PRIME - PSI THETA
C
      MMF=NT(6)
      DO 7300 MM=1,MMF
      JJ=MM+NTS5
      IF(H)7202,7300,7202

```

7202 AS(II,JJ)=C46\*FR144(KK,MM)+C461\*FE44(KK,MM)+C462\*FREE44(KK,MM)  
7300 CONTINUE

C  
C  
C

SUBMATRIX PSI THETA PRIME - PSI X

MMF=NT(7)  
DO 7100 MM=1,MMF  
JJ=MM+NTS6  
IF(H)7012,7100,7012  
7012 IF(N)7013,7100,7013  
7013 AS(II,JJ)=C47\*FR154(KK,MM)-C471\*FE54(KK,MM)  
7100 CONTINUE  
GOTO(10028,20028),NCHECK  
10028 CALL CHECK (28)  
20028 CONTINUE  
7000 CONTINUE

C  
C  
C

FIFTH SET OF ROWS

KKF=NT(5)  
DO 5000 KK=1,KKF  
II=KK+NTS4  
IF(KI(5))5002,5001,5002  
5001 K=KK-1  
GO TO 5003  
5002 K=KK  
5003 CONTINUE

C  
C  
C

SUBMATRIX PSI X PRIME - W

MMF=NT(3)  
DO 5100 MM=1,MMF  
JJ=MM+NTS2  
IF(MI(3))5012,5011,5012  
5011 M=MN-1  
GO TO 5013

```

5012 M=MM
5013 GO TO (5014,5015),ICALC
5014 FRE35(KK,MM)=IRE35(K,M)
5015 AS(II,JJ)=-C351*FRE35(KK,MM)
5100 CONTINUE
      GOTO(10022,20022),NCHECK
10022 CALL CHECK (22)
20022 CONTINUE
C
C      SUBMATRIX PSI X PRIME - PSI THETA PRIME
C
      MMF=NT(4)
      DO 5500 MM=1,MMF
      JJ=MM+NTS3
      IF(N)5402,5500,5402
5402 IF(MI(4))5404,5403,5404
5403 M=MM-1
      GO TO 5405
5404 M=MM
5405 GO TO (5406,5407),ICALC
5406 FE45(KK,MM)=IE45(K,M)
5407 AS(II,JJ)=C45*FR154(MM,KK)+C451*FE45(KK,MM)
5500 CONTINUE
      GOTO(10023,20023),NCHECK
10023 CALL CHECK (23)
20023 CONTINUE
C
C      SUBMATRIX PSI X PRIME - PSI X PRIME
C
      MMF=NT(5)
      DO 5300 MM=1,MMF
      JJ=MM+NTS4
      IF(MI(5))5202,5201,5202
5201 M=MM-1
      GO TO 5203
5202 M=MM

```

```

5203 GO TO (5204,5205),ICALC
5204 FR155(KK,MM)=IR155(K,M)
      FR55(KK,MM)=IR55(K,M)
      FE55(KK,MM)=IE55(K,M)
      FREE55(KK,MM)=IREE55(K,M)
5205 AI(II,JJ)=CJF*FR55(KK,MM)
      OAS(II,JJ)=C55*FR155(KK,MM)+C550*FR55(KK,MM)+C551*FE55(KK,MM)
      1 +C552*FREE55(KK,MM)
5300 CONTINUE
      GOTO(10024,20024),NCHECK
10024 CALL CHECK (24)
20024 CONTINUE
C
C SUBMATRIX PSI X PRIME - PSI THETA
C
      MMF=NT(6)
      DO 5400 MM=1,MMF
      JJ=MM+NTS5
      IF(N)5301,5400,5301
5301 IF(H)5303,5400,5303
5303 AS(II,JJ)=C56*FR154(MM,KK)+C561*FE45(KK,MM)
5400 CONTINUE
      GOTO(10025,20025),NCHECK
10025 CALL CHECK (25)
20025 CONTINUE
C
C SUBMATRIX PSI X PRIME - PSI X
C
      MMF=NT(7)
      DO 5200 MM=1,MMF
      JJ=MM+NTS6
      IF(H)5102,5200,5102
5102 AS(II,JJ)=C57*FR155(KK,MM)+C571*FE55(KK,MM)
      1 +C572*FREE55(KK,MM)
5200 CONTINUE
      GOTO(10026,20026),NCHECK

```

```

10026 CALL CHECK (26)
20026 CONTINUE
5000 CONTINUE
C
C   SIXTH SET OF ROWS
C
      KKF=NT(6)
      IF(H)6003,6000,6003
6003 DO 6000 KK=1,KKF
      II=KK+NTS5
C
C   SUBMATRIX PSI THETA - V
C
      MMF=NT(2)
      DO 6100 MM=1,MMF
      JJ=MM+NT(1)
      AS(II,JJ)=C26*F42(MM,KK)
6100 CONTINUE
C
C   SUBMATRIX PSI THETA - W
C
      MMF=NT(3)
      DO 6200 MM=1,MMF
      JJ=MM+NTS2
      IF(N)6102,6200,6102
6102 AS(II,JJ)=C36*F43(MM,KK)
6200 CONTINUE
C
C   SUBMATRIX PSI THETA - PSI THETA PRIME
C
      MMF=NT(4)
      DO 6600 MM=1,MMF
      JJ=MM+NTS3
      AS(II,JJ)=C46*FR144(KK,MM)+C461*FE44(KK,MM)+C462*FREE44(KK,MM)
6600 CONTINUE
C

```

C SUBMATRIX PSI THETA - PSI X PRIME

C

MMF=NT(5)  
DO 6400 MM=1,MMF  
JJ=MM+NTS4  
IF(N)6302,6400,6302  
6302 AS(II,JJ)=C56\*FR154(MM,KK)-C561\*FE54(KK,MM)  
6400 CONTINUE

C

C SUBMATRIX PSI THETA - PSI THETA

C

MMF=NT(6)  
DO 6500 MM=1,MMF  
JJ=MM+NTS5  
AI(II,JJ)=CJC\*FR44(KK,MM)  
OAS(II,JJ)=C66\*FR144(KK,MM)+C660\*FR44(KK,MM)+C661\*FE44(KK,MM)  
1 +C662\*FREE44(KK,MM)  
6500 CONTINUE

C

C SUBMATRIX PSI THETA - PSI X

C

MMF=NT(7)  
DO 6300 MM=1,MMF  
JJ=MM+NTS6  
IF(N)6202,6300,6202  
6202 AS(II,JJ)=C67\*FR154(KK,MM)-C671\*FE54(KK,MM)  
6300 CONTINUE  
6000 CONTINUE  
GOTO(10027,20027),NCHECK  
10027 CALL CHECK (27)  
20027 CONTINUE

C

C SEVENTH SET OF ROWS

C

KKF=NT(7)  
IF(H)4003,4000,4003

```

4003 DO 4000 KK=1,KKF
      II=KK+NTS6
C
C   SUBMATRIX PSI X - W
C
      MMF=NT(3)
      DO 4100 MM=1,MMF
        JJ=MM+NTS2
        AS(II,JJ)=-C371*FRE35(KK,MM)
4100 CONTINUE
      GOTO(10017,20017),NCHECK
10017 CALL CHECK (17)
20017 CONTINUE
C
C   SUBMATRIX PSI X - PSI THETA PRIME
C
      MMF=NT(4)
      DO 4500 MM=1,MMF
        JJ=MM+NTS3
        IF(N)4402,4500,4402
4402 AS(II,JJ)=C47*FR154(MM,KK)+C471*FE45(KK,MM)
4500 CONTINUE
      GOTO(10018,20018),NCHECK
10018 CALL CHECK (18)
20018 CONTINUE
C
C   SUBMATRIX PSI X - PSI X PRIME
C
      MMF=NT(5)
      DO 4300 MM=1,MMF
        JJ=MM+NTS4
        AS(II,JJ)=C57*FR155(KK,MM)+C571*FE55(KK,MM)
        +C572*FREE55(KK,MM)
4300 CONTINUE
      GOTO(10019,20019),NCHECK
10019 CALL CHECK (19)

```

20019 CONTINUE

C

C SUBMATRIX PSI X - PSI THETA

C

MMF=NT(6)

DO 4400 MM=1,MMF

JJ=MM+NTS5

IF(N)4302,4400,4302

4302 AS(II,JJ)=C67\*FR154(MM,KK)+C671\*FE45(KK,MM)

4400 CONTINUE

GOTO(10020,20020),NCHECK

10020 CALL CHECK (20)

20020 CONTINUE

C

C SUBMATRIX PSI X - PSI X

C

MMF=NT(7)

DO 4200 MM=1,MMF

JJ=MM+NTS6

AI(II,JJ)=CJC\*FR55(KK,MM)

OAS(II,JJ)=C77\*FR155(KK,MM)+C770\*FR55(KK,MM)+C771\*FE55(KK,MM)

I +C772\*FREE55(KK,MM)

4200 CONTINUE

GOTO(10021,20021),NCHECK

10021 CALL CHECK (21)

20021 CONTINUE

4000 CONTINUE

C

C WRITE ETA'S, C'S, AND INTEGRALS, IF DESIRED.

C

GO TO (30001,40001),NTEST

30001 WRITE(3,50001) N1,N2,N3,N4,N5,N6,N7,N8,N9,N10,N11,N12,N13,N14,

I N15,N16

50001 FORMAT ('0',8E15.4/1X,8E15.4)

WRITE (3,50002) C11,CM,C111,C112,C12,C121,C13,C131,C22,C221,C222,

I C23,C26,C33,C331,C332,C34,C35,C351,C36,C44,C440,CJC,C441,

```

2 C442,C45,C451,          C46,C461,C47,C471,C55,C550,CJF,C551,
3 C552,C56,C561,C57,C571,C66,C660,C661,C662,C67,C671,C77,C771,C772
50002 FORMAT ('0',8E15.4/6(1X,8E15.4/))
      GO TO (50004,40001),ICALC
50004 WRITE(3,50003)
      1      FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,
      2FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,
      3FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,
      4FE55,FREE55
50003 FORMAT('0',6D15.4/203(1X,6D15.4/))
40001 CONTINUE
C
C   FOR A HOMOGENEOUS SHELL (H=0), THE ROWS AND COLUMNS CORRESPONDING
C   TO THE CORE ROTATIONS ARE REMOVED, AND THE STIFFNESS AND INERTIA
C   MATRICES ARE COMPRESSED ACCORDINGLY.
C
      IF(H)8030,8000,8030
8000 NTS7=NTS7-NT(6)-NT(7)
      NT(6)=0
      NT(7)=0
      GOTO(10029,20029),NCHECK
10029 CALL CHECK (29)
20029 CONTINUE
8030 CONTINUE
C
C   PRINT THE STIFFNESS MATRIX
C
      GOTO(120,140),NWRITE
120 WRITE(3,121)
121 FORMAT('0',55X,'STIFFNESS MATRIX')
      DO 128 I=1,NTS7
      J1=1
      J7=7
      JTAG=1
122 WRITE(3,123)(J,J=J1,J7)
123 FORMAT('0',10X,'COL',I3,6(10X,'COL',I3))

```

```

        WRITE(3,124)I,(AS(I,J),J=J1,J7)
124  FORMAT(' ROW',I3,7D16.8)
        GOTO(125,128),JTAG
125  J1=J1+7
        IF(J7+7-NTS7)126,127,127
126  J7=J7+7
        GO TO 122
127  J7=NTS7
        JTAG=2
        GO TO 122
128  CONTINUE

```

C  
C  
C

```

        PRINT THE INERTIA MATRIX

        WRITE(3,129)
129  FORMAT('0',46X,'BLOCK DIAGONAL OF INERTIA MATRIX')
        ICT=1
        IO=1
        IL=NT(1)
        KTAG=1
130  DO 133 I=IO,IL
131  WRITE(3,132)(J,J=IO,IL)
132  FORMAT('0',10X,'COL',I3,5(10X,'COL',I3))
133  WRITE(3,134)I,(AI(I,J),J=IO,IL)
134  FORMAT(' ROW',I3,7D16.8)
        GOTO(135,140),KTAG
135  IO=IO+NT(ICT)
        IL=IL+NT(ICT+1)
        ICT=ICT+1
        IF(H)138,136,138
136  IF(ICT-5)130,137,137
137  KTAG=2
        GO TO 130
138  IF(ICT-7)130,139,139
139  KTAG=2
        GO TO 130

```

```

140 CONTINUE
C
C   WRITE THE MATRICES ON TAPE
C
      GO TO (200,250),NTAPE
200 GO TO (201,202),ITAPE
201 ITAPE=2
      REWIND 7
      REWIND 8
202 CONTINUE
      WRITE(7) NAME,BCOND,ANG,RO,XL,T,H,MUX,MUT
      WRITE(7) EX,ET,GZXF,GTZF,GXTF,GZXC,GTZC
      WRITE(7) RF,KXF,KTF,RC,KXC,KTC,NT,N,SA
      WRITE(7) IEG,IVEC,IDET,MIT,MITS,ALRS,GBR,IQUIT
      GOTO(10030,20030),NCHECK
10030 CALL CHECK (30)
20030 CONTINUE
      250 CONTINUE
C
C   TESTS: WHERE DO I GO FROM HERE?
C
900 IF(N-NMAX)902,903,903
902 IGO=1
      ICALC=2
      GO TO 907
903 GO TO (905,904),NTERM
904 IGO=2
      GO TO 907
905 GO TO (908,906),NCASE
906 IGO=3
907 NSTOP=1
      GO TO 909
908 NSTOP=2
      IGO=4
909 GO TO (910,912),NTAPE
910 WRITE (8) NTAGBC,NTS7,AS,AI,NSTOP,EX

```

```
912 CONTINUE
913 GO TO (10,7,1,914),IGO
914 RETURN
END
```

```
      SUBROUTINE CHECK (I)
      WRITE (3,1) I
1  FORMAT (' CHECK ',I3)
      RETURN
      END
```

```
      SUBROUTINE PART2
C  INVERSION OF STIFFNESS MATRIX AND MULTIPLICATION TIMES
C  INERTIA MATRIX
      DOUBLE PRECISION BIG,X,EX,DAS,DAI,LIT,NORM,Y,FACTOR
      DOUBLE PRECISION AS(42,42),AI(42,42),A(42,42),ASV(1764),DET
      DIMENSION LMINV(42),MMINV(42)
      ITAPE=1
200 GO TO (201,202),ITAPE
201 ITAPE=2
      REWIND 8
      REWIND 9
202 CONTINUE
      READ (8) NTAGBC,NTS7,AS,AI,NSTOP,EX
      NN=NTS7
C
C  NORMALIZE THE MATRICES
C
      DO 1 I=1,NN
      DO 1 J=1,NN
1  A(I,J)=0.DO
C
```

C FIND THE LARGEST ELEMENT IN (AS) OR (AI)  
C

```
BIG=0.DO  
LIT=AI(1,1)  
DO 28 I=1,NN  
DAS=DABS(AS(I,I))  
DAI=DABS(AI(I,I))  
IF(DAS-DAI)2,3,3  
2 X=DAI  
Y=DAS  
GO TO 4  
3 X=DAS  
Y=DAI  
4 IF(BIG-X)5,6,6  
5 BIG=X  
6 IF(LIT-Y)28,28,27  
27 LIT=Y  
28 CONTINUE  
NORM=DSQRT(BIG/LIT)  
BIG=BIG/NORM  
GO TO (9,7),NTAGBC  
7 DO 8 I=1,NN  
DO 8 J=1,NN  
A(I,J)=AS(I,J)  
AS(I,J)=AI(I,J)/BIG  
8 AI(I,J)=A(I,J)/BIG  
GO TO 11  
9 DO 10 I=1,NN  
DO 10 J=1,NN  
AS(I,J)=AS(I,J)/BIG  
10 AI(I,J)=AI(I,J)/BIG  
11 CONTINUE
```

C  
C FIND THE FLEXIBILITY MATRIX BY INVERTING THE STIFFNESS MATRIX  
C  
CALL DARRAY (2,NN,NN,42,42,ASV,AS)

```

      CALL DMINV (ASV, NN, DET, LMINV, MMINV)
      CALL FARRAY (1, NN, NN, 42, 42, ASV, AS)
C (AS) IS NOW (AS)-INVERTED
C
C   MULTIPLY THE FLEXIBILITY MATRIX TIMES THE INERTIA MATRIX
C
      CALL MULTM1 (AS, AI, A, NN, NN, NN)
C
C   NORMALIZE BEFORE GOING TO EIGENVALUE SOLUTION.
C
      BIG=DABS(A(1,1))
      LIT=BIG
      DO 33 I=2, NN
      DAS=DABS(A(I,I))
      IF(BIG-DAS)30,31,31
30  BIG=DAS
      GO TO 33
31  IF(LIT-DAS)33,33,32
32  LIT=DAS
33  CONTINUE
      NORM=DSQRT(BIG/LIT)
      FACTOR=BIG/NORM
      DO 40 I=1, NN
      DO 40 J=1, NN
40  A(I,J)=A(I,J)/FACTOR
      WRITE(9) NTAGBC, NTS7, A, NSTOP, FACTOR
      GO TO (200, 999), NSTOP
999 RETURN
      END

```

```

      SUBROUTINE DARRAY (MODE, I, J, N, M, S, D)
C SEE WRITE-UP IN IBM SSP, PAGE 85
      DOUBLE PRECISION S(1), D(1)
      NI=N-I

```

```

      IF(MODE-1) 100, 100, 120
100  IJ=I*J+1
      NM=N*J+1
      DO 110 K=1,J
      NM=NM-NI
      DO 110 L=1,I
      IJ=IJ-1
      NM=NM-1
110  D(NM)=S(IJ)
      GO TO 140
120  IJ=0
      NM=0
      DO 130 K=1,J
      DO 125 L=1,I
      IJ=IJ+1
      NM=NM+1
125  S(IJ)=D(NM)
130  NM=NM+NI
140  RETURN
      END

```

```

C  SUBROUTINE DMINV(A,N,D,L,M)
    SEE WRITE-UP IN IBM SSP.
    DIMENSION A(1)
    DIMENSION L(1),M(1)
    DOUBLE PRECISION A,D,BIGA,HOLD
    D=1.0D+00
    NK=-N
    DO 80 K=1,N
    NK=NK+N
    L(K)=K
    M(K)=K
    KK=NK+K
    BIGA=A(KK)

```

MINV 033

MINV 042  
 MINV 056  
 MINV 057  
 MINV 058  
 MINV 059  
 MINV 060  
 MINV 061  
 MINV 062  
 MINV 063

	DO 20 J=K,N	MINV 064
	IZ=N*(J-1)	MINV 065
	DO 20 I=K,N	MINV 066
	IJ=IZ+I	MINV 067
	10 IF(DABS(BIGA)-DABS(A(IJ))) 15,20,20	MINV 068
	15 BIGA=A(IJ)	MINV 069
	L(K)=I	MINV 070
	M(K)=J	MINV 071
	20 CONTINUE	MINV 072
C		MINV 073
C	INTERCHANGE ROWS	MINV 074
C		MINV 075
	J=L(K)	MINV 076
	IF(J-K) 35,35,25	MINV 077
	25 KI=K-N	MINV 078
	DO 30 I=1,N	MINV 079
	KI=KI+N	MINV 080
	HOLD=-A(KI)	MINV 081
	JI=KI-K+J	MINV 082
	A(KI)=A(JI)	MINV 083
	30 A(JI) =HOLD	MINV 084
C		MINV 085
C	INTERCHANGE COLUMNS	MINV 086
C		MINV 087
	35 I=M(K)	MINV 088
	IF(I-K) 45,45,38	MINV 089
	38 JP=N*(I-1)	MINV 090
	DO 40 J=1,N	MINV 091
	JK=NK+J	MINV 092
	JJ=JP+J	MINV 093
	HOLD=-A(JK)	MINV 094
	A(JK)=A(JI)	MINV 095
	40 A(JI) =HOLD	MINV 096
C		MINV 097
C	DIVIDE COLUMN BY MINUS PIVOT (VALUE OF PIVOT ELEMENT IS	MINV 098
C	CONTAINED IN BIGA)	MINV 099

```

C
45 IF(BIGA) 48,46,48
46 D=0.0D+00
   RETURN
48 DO 55 I=1,N
   IF(I-K) 50,55,50
50 IK=NK+I
   A(IK)=A(IK)/(-BIGA)
55 CONTINUE

C
C   REDUCE MATRIX
C
DO 65 I=1,N
IK=NK+I
IJ=I-N
DO 65 J=1,N
IJ=IJ+N
IF(I-K) 60,65,60
60 IF(J-K) 62,65,62
62 KJ=IJ-I+K
   A(IJ)=A(IK)*A(KJ)+A(IJ)
65 CONTINUE

C
C   DIVIDE ROW BY PIVOT
C
KJ=K-N
DO 75 J=1,N
KJ=KJ+N
IF(J-K) 70,75,70
70 A(KJ)=A(KJ)/BIGA
75 CONTINUE

C
C   PRODUCT OF PIVOTS
C
C   REMOVED D=D*BIGA
C

```

```

MINV 100
MINV 101
MINV 102
MINV 103
MINV 104
MINV 105
MINV 106
MINV 107
MINV 108
MINV 109
MINV 110
MINV 111
MINV 112
MINV 113
MINV 114
MINV 115
MINV 116
MINV 117
MINV 118
MINV 119
MINV 120
MINV 121
MINV 122
MINV 123
MINV 124
MINV 125
MINV 126
MINV 127
MINV 128
MINV 129
MINV 130
MINV 131
MINV 132
MINV 133
MINV 135

```

C	REPLACE PIVOT BY RECIPROCAL	MINV 136
C	A(KK)=1.0D+00/BIGA	MINV 137
	80 CONTINUE	MINV 138
C	FINAL ROW AND COLUMN INTERCHANGE	MINV 139
C		MINV 140
	K=N	MINV 141
100	K=(K-1)	MINV 142
	IF(K) 150,150,105	MINV 143
105	I=L(K)	MINV 144
	IF(I-K) 120,120,108	MINV 145
108	JQ=N*(K-1)	MINV 146
	JR=N*(I-1)	MINV 147
	DO 110 J=1,N	MINV 148
	JK=JQ+J	MINV 149
	HOLD=A(JK)	MINV 150
	JJ=JR+J	MINV 151
	A(JK)=-A(JJ)	MINV 152
110	A(JJ) =HOLD	MINV 153
120	J=M(K)	MINV 154
	IF(J-K) 100,100,125	MINV 155
125	KI=K-N	MINV 156
	DO 130 I=1,N	MINV 157
	KI=KI+N	MINV 158
	HOLD=A(KI)	MINV 159
	JJ=KI-K+J	MINV 160
	A(KI)=-A(JJ)	MINV 161
130	A(JJ) =HOLD	MINV 162
	GO TO 100	MINV 163
150	RETURN	MINV 164
	END	MINV 165
		MINV 166
		MINV 167

SUBROUTINE FARRAY (MODE, I, J, N, M, S, D) :

C THIS SUBROUTINE IS IDENTICAL TO DARRAY.

```
DOUBLE PRECISION S(1),D(1)
NI=N-I
IF(MODE-1) 100, 100, 120
100 IJ=I*J+1
    NM=N*J+1
    DO 110 K=1,J
        NM=NM-NI
        DO 110 L=1,I
            IJ=IJ-1
            NM=NM-1
110 D(NM)=S(IJ)
    GO TO 140
120 IJ=0
    NM=0
    DO 130 K=1,J
        DO 125 L=1,I
            IJ=IJ+1
            NM=NM+1
125 S(IJ)=D(NM)
130 NM=NM+NI
140 RETURN
END
```

```
SUBROUTINE MULTM1 (A,B,C,L,M,N)
DOUBLE PRECISION A(42,42),B(42,42),C(42,42)
DO 999 I=1,L
DO 999 J=1,N
C(I,J) = 0.D+00
DO 999 K=1,M
999 C(I,J) = C(I,J)+A(I,K)*B(K,J)
RETURN
END
```

```

SUBROUTINE PART3
C   CALCULATION OF NATURAL FREQUENCIES & NORMAL MODE SHAPES
DOUBLE PRECISION A(42,42),EVEC(42,42),EVAL(42)
DOUBLE PRECISION ALRS,GBR
DOUBLE PRECISION EX,COMEGA,FACTOR
COMMON EVEC
ITAPE=1
200 GO TO (201,202),ITAPE
201 ITAPE=2
REWIND 7
REWIND 9
202 CONTINUE
CALL WRITE1 (EX,COMEGA)
READ (7) IEG,IVEC,IDET,MIT,MITS,ALRS,GBR,IQUIT
READ (9) NTAGBC,NTS7,A,NSTOP,FACTOR
NN=NTS7

C
C   IF IEG = 1, EVERY ITERATION OF THE EIGENVALUE IS PRINTED.
C   OTHERWISE, IEG = 0.
C   IF IVEC = 1, THE EIGENVECTOR IS CALCULATED AND PRINTED.
C   OTHERWISE, IVEC = 0.
C   SET IDET = 1
C   ALRS IS THE INITIAL EIGENVALUE GUESS.
C   GBR IS THE INCREMENT TAKEN WHEN THE INVERSE POWER METHOD IS USED.
C   MIT IS THE MAXIMUM NUMBER OF ITERATIONS TAKEN FOR THE DIRECT
C   POWER METHOD.
C   MITS IS THE MAXIMUM NUMBER OF ITERATIONS TAKEN FOR THE INVERSE
C   POWER METHOD.
C
CALL MATSUB (NN,IEG,IVEC,ALRS,GBR,IDET,MIT,MITS,IQUIT,NTAGBC,
1 A,EVAL)
CALL WRITE2 (NN,NTAGBC,EX,COMEGA,EVAL,FACTOR,IQUIT)
GO TO (200,999),NSTOP
999 RETURN

```

END

```
      SUBROUTINE WRITE1 (EX, COMEGA)
      DOUBLE PRECISION NAME(10), BCOND(3)
      ODOUBLE PRECISION ANG, RO, XL, T, H, MUX, MUT, EX, ET, GZXF, GTZF, GXTF, GZXC,
      1 GTZC, RF, KXF, KTF, RC, KXC, KTC, COMEGA, SA
      DIMENSION NT(7)
      READ (7) NAME, BCOND, ANG, RO, XL, T, H, MUX, MUT
      READ (7) EX, ET, GZXF, GTZF, GXTF, GZXC, GTZC
      READ (7) RF, KXF, KTF, RC, KXC, KTC, NT, N, SA
100  WRITE(3, 101)(NAME(I), I=1, 10)
101  FORMAT('1', 23X, 10A8)
      WRITE(3, 102)(BCOND(I), I=1, 3)
102  FORMAT('0', 42X, 'BOUNDARY CONDITIONS -- ', 3A8)
      WRITE(3, 103)
103  FORMAT('0', 56X, 'SHELL GEOMETRY')
      WRITE(3, 104)ANG, RO, XL, T, H
104  FORMAT(' ', 51X, 'ALPHA = ', F6.2, ' DEGREES'/53X, 'RO = ', F8.3, ' INCHE
      1S' /54X, 'L = ', F8.3, ' INCHES'/54X, 'T = ', F8.4, ' INCHES'/54X,
      2 'H = ', F8.4, ' INCHES')
      WRITE(3, 105)
105  FORMAT('0', 53X, 'MATERIAL PROPERTIES'//60X, 'FACINGS')
      WRITE(3, 106)EX, MUX, ET, MUT, GZXF, KXF, GTZF, KTF, GXTF, RF
106  FORMAT(' ', 35X, 'EX = ', D13.6, ' PSI.', 7X, 'MUX = ', F6.3/36X,
      1 'ET = ', D13.6, ' PSI.', 7X, 'MUT = ', F6.3/36X, 'GZX = ', D13.6,
      2 ' PSI.', 7X, 'KX = ', F6.3/36X, 'GTZ = ', D13.6, ' PSI.', 7X, 'KT = ',
      3 F6.3/36X, 'GXT = ', D13.6, ' PSI.', 7X, 'RHO = ', D13.6,
      4 ' LB-SEC**2/IN**4')
      IF(H) 109, 107, 109
107  WRITE(3, 108)
108  FORMAT('0', 50X, 'HOMOGENEOUS SHELL, NO CORE')
      GO TO 111
109  WRITE(3, 110)GZXC, KXC, GTZC, KTC, RC
110  FORMAT('0', 61X, 'CORE' / 36X, 'GZX = ', D13.6, ' PSI.', 7X, 'KX = ',
```

```

1 F6.3/ 36X, 'GTZ = ', D13.6, ' PSI.', 7X, 'KT = ', F6.3/46X, 'RHO = ',
2 D13.6, ' LB-SEC**2/IN**4'
111 CONTINUE
114 WRITE(3,115)(I,NT(I),I=1,7)
115 FORMAT('0', 45X, 'NUMBER OF TERMS IN SERIES (' ,I1,') =',I2//,
1 (46X, 'NUMBER OF TERMS IN SERIES (' ,I1,') =',I2//)
117 WRITE(3,118) N
118 FORMAT('0',62X, 'N =',I3)
COMEGA=(RO+XL*SA)*DSQRT(RF*(1.DO-MUX*MUT)/EX)
RETURN
END

```

```

OSUBROUTINE MATSUB (M,IEG,IVEC,ALRS,GBR,IDET,MIT,MITS,IQUIT,
1 NTAGBC,CR,ZR)
DOUBLE PRECISION CR(42,42),ZR(42),VALUR(42,42),YR(42),XR(42),
1 AR(42,42),BR(42,42)
DOUBLE PRECISION SUMR,PRDR,TRACER,DETR,T1,ALR,ALRS,BIG,AM,RQNR,
1 RQD,AMUR,AMM,ALRC,TS,EP1,FM,SM,RR,SMALL,SR,SUM,EP2,GBR,ZLAG,
2 ZLIT,ZMAGT
COMMON VALUR
IAARD=M+1
IONE=1
ITWO=2
N=M
SUMR=0.0
PRDR=1.0
TRACER=0.0
DO 450 I=1,N
450 TRACER=TRACER+CR(I,I)
C SET UP MATRICES
DO 519 I=1,N
DO 519 J=1,N
BR(I,J)=CR(I,J)
519 AR(I,J)=CR(I,J)

```

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```

C	EVALUATE DETERMINENT	024
	IA=1	
	ID=1	
	MM=M	027
	INTER=0	028
	GO TO 535	029
520	DETR=1.00	
	INTER= MOD (INTER,2)	
	IF (INTER) 1000,917,810	037
1000	RETURN	
810	DETR=-DETR	039
917	GO TO (811,912),ID	
811	CONTINUE	
	ID=2	
	IA=2	
	IB=1	
	IC=1	
	ISL=-1	049
	GO TO 92	050
523	ISL=0	051
C	EIGENVALUE GUESS OR ORIGIN TRANSLATION	052
9	ALR=ALRS	053
	IT=1	055
C	EIGENVECTOR GUESS	056
403	DO 504 I=1,N	057
504	XR(I)=1.0	
4	DO 5 I=1,N	060
5	AR(I,I)=AR(I,I)-ALR	
C	FIRST ITERATION - POWER METHOD	063
	IJ=1	064
10	BIG=0.	065
C	COMPUTE Y=(A-ALPHA)*X	066
	DO 13 I=1,N	067
	YR(I)=0.	068
	DO 11 J=1,N	070
11	YR(I)=YR(I)+AR(I,J)*XR(J)	

	AM=YR(I)**2	
	IF (AM-BIG) 13,13,12	074
12	BIG=AM	075
	JJ=I	076
13	CONTINUE	077
	IF (BIG) 109,106,109	078
C	EXACT EIGENVALUE AND EIGENVECTOR - Y=0. FLAG=1000	079
106	ICT=1000	080
	DO 108 I=1,N	081
	JJ=I	082
	IF (XR(I)-1.0) 108,118,108	083
118	ISL=1	084
	GO TO 92	085
108	CONTINUE	086
	WRITE(3,650)	
650	FORMAT (48H ERROR. EIGENVECTOR NOT NORMALIZED IN METHOD 1.)	088
	GO TO 990	089
C	MU RAYLEIGH QUOTIENT - (Y,X)/(X,X)=MU	090
109	RQNR=0.	091
	RQD=0.	093
	DO 14 I=1,N	094
	RQNR=RQNR+XR(I)*YR(I)	
14	RQD=RQD+XR(I)**2	
	IF (RQD) 10001,10000,10001	
10000	AMUR = 0.DO	
	GO TO 10002	
10001	CONTINUE	
	AMUR=RQNR/RQD	098
10002	CONTINUE	
	AMM=AMUR**2	
	IF (IEG) 1000,81,80	101
80	ALRC=AMUR+ALR	102
C	TEST FIRST ITERATION	106
C	MAGNITUDE OF (Y-MU*X)=TS	107
81	TS=0.	108
	DO 15 I=1,N	109

	150	TS=TS+(YR(I)-AMUR*XR(I))**2	
C		NORMALIZATION	112
		DO 16 I=1,N	113
	16	XR(I)=(YR(JJ)*YR(I))/BIG	
		XR(JJ)=1.0	116
		EP1=AMUR*1.0-3	
		IF (RQD) 111,20,111	
111		IF (TS/RQD-EP1) 20,20,18	118
	18	IF (IJ-MIT) 19,20,20	119
	19	IJ=IJ+1	120
		GO TO 10	121
C		SECOND ITERATION - INVERSE POWER METHOD	122
	20	ICT=IJ	123
		MIT2=MITS+IJ	124
		ALR=AMUR+ALR	125
		MM=N	127
		DO 310 I=1,N	128
310		AR(I,I)=AR(I,I)-AMUR	
		GO TO 29	131
	99	DO 100 I=1,N	132
100		AR(I,I)=AR(I,I)-ALR	
	29	IJ=IJ+1	135
C		GAUSSIAN ELIMINATION - (A-ALPHA)*Y=X	136
	535	DO 27 I=2,MM	137
		IM1=I-1	138
		DO 27 J=1,IM1	139
	21	FM=AR(I,J)*AR(I,J)	
		SM=AR(J,J)*AR(J,J)	
		IF (FM-SM) 24,24,22	142
C		ROW INTERCHANGE - IF NECESSARY	143
	22	DO 23 K=J,MM	144
		T1=AR(J,K)	145
		AR(J,K)=AR(I,K)	147
	23	AR(I,K)=T1	
		T1=XR(J)	151
		XR(J)=XR(I)	153

	XR(I)=T1	155
	T1=FM	157
	FM=SM	158
	SM=T1	159
	INTER=INTER+1	160
	24 IF (SM) 25,27,25	161
	25 IF (FM) 90,27,90	162
C	TRIANGULARIZATION	163
	90 RR=(AR(I,J)*AR(J,J))/SM	
	DO 26 K=J,MM	166
	26 AR(I,K)=AR(I,K)-RR*AR(J,K)	
	AR(I,J)=0.	169
	XR(I)=XR(I)-RR*XR(J)	
	27 CONTINUE	173
	GO TO (520,530,911,530),IA	
530	SMALL=1000.	175
	DO 28 K=1,MM	176
	IKK=K	177
	T1=AR(K,K)**2	
	IF (T1) 750,752,750	179
750	IF (T1-SMALL) 751,28,28	180
751	SMALL=T1	181
	IZ=K	182
	28 CONTINUE	183
	GO TO (40,753,40),IB	
752	IZ=IKK	185
	IF (ISL) 753,30,30	186
C	EXACT EIGENVALUE - (A-ALPHA) SINGULAR. FLAG=2000	187
	30 ISL=1	188
	ICT=2000	189
	DO 974 I=1,MM	190
974	XR(I)=0.0	
753	YR(IZ)=1.0	193
	JJ=IZ	195
	BIG=1.0	196
	IF (IZ-MM) 33,32,33	197

32	IZZ=2	198
	GO TO 95	199
33	IZZ=IZ+1	200
	DO 31 I=IZZ,MM	201
31	YR(I)=0.	
	IZZ=MM-IZ+2	204
	IF (IZ-1) 95,49,95	205
C	BACKWARD SUBSTITUTION	206
40	IZZ=1	207
41	BIG=0.	208
95	DO 46 I=IZZ,MM	209
	II=MM-I+1	210
	KK=II+1	211
	SR=0.	212
	IF (I-1) 42,44,42	214
42	DO 43 K=KK,MM	215
43	SR=SR+AR(II,K)*YR(K)	
44	T1=AR(II,II)**2	
	YR(II)=(AR(II,II)*(XR(II)-SR))/T1	
	AM=YR(II)**2	
	IF (AM-BIG) 46,46,45	222
45	JJ=II	223
	BIG=AM	224
46	CONTINUE	225
C	NORMALIZATION - X=NORMALIZED Y	226
49	DO 47 I=1,MM	227
47	XR(I)=(YR(JJ)*YR(I))/BIG	
	XR(JJ)=1.0	230
92	DO 601 I=1,N	232
	DO 601 J=1,N	233
601	AR(I,J)=BR(I,J)	
116	IF (ISL) 755,50,60	236
755	GO TO (523,704,525),IC	
C	ALPHA RAYLEIGH QUOTIENT - (AX,X)/(X,X)=ALPHA	238
50	ALR=0.	239
	SUM=0.0	241

55 DO 52 I=1,N	242
YR(I)=0.	243
DO 51 K=1,N	245
51 YR(I)=YR(I)+AR(I,K)*XR(K)	
ALR=ALR+XR(I)*YR(I)	
52 SUM=SUM+XR(I)*XR(I)	
IF (SUM) 20001,20000,20001	
20000 ALR = 0.DO	
GO TO 20002	
20001 CONTINUE	
ALR=ALR/SUM	251
20002 CONTINUE	
AM=ALR**2	
IF (IEG) 1000,83,82	254
82 CONTINUE	
C TEST SECOND ITERATION	256
83 TS=0.	257
DO 53 I=1,N	258
T1=YR(I)-ALR*XR(I)	
53 TS=TS+T1**2	
EP2=ALR*1.D-8	
IF (SUM) 93,60,93	
93 IF (TS/SUM-EP2) 60,60,301	262
301 IF (IJ-MIT2) 99,400,400	263
400 WRITE(3,401) IT	
401 FORMAT (54H INVERSE POWER METHOD NOT CONVERGED ON TRY NUMBER	265
1I5)	266
IF (IT-3) 402,990,402	267
402 ALR=ALR+GBR	268
IT=IT+1	270
WRITE(3,820) ALR	
820 FORMAT (11H ALPHA= E20.8)	
GO TO 4	273
60 ISL=0	274
63 WRITE(3,64) N,ALR,EP1,EP2,IGT,IJ	
64 FORMAT(15,15H TH EIGENVALUE= E18.8,20X,2E10.2,10X,2I5)	

	ZR(N)=ALR	276A
	SUMR=SUMR+ALR	277
	T1=PRDR*ALR	
	PRDR=T1	281
C	DEFLATION OF MATRIX	282
	IF (JJ-N) 61,65,61	283
C	PERMUTATION OPERATION	284
61	T1=XR(JJ)	285
	XR(JJ)=XR(N)	287
	XR(N)=T1	289
	DO 68 K=1,N	291
	T1=AR(JJ,K)	292
	AR(JJ,K)=AR(N,K)	294
68	AR(N,K)=T1	
	DO 62 K=1,N	298
	T1=AR(K,JJ)	299
	AR(K,JJ)=AR(K,N)	301
62	AR(K,N)=T1	
C	DEFLATION	305
65	N=N-1	306
	DO 66 I=1,N	307
	DO 66 J=1,N	308
66	AR(I,J)=AR(I,J)-XR(I)*AR(N+1,J)	
	DO 600 I=1,N	311
	DO 600 J=1,N	312
600	BR(I,J)=AR(I,J)	
C	COMPUTE EIGENVECTOR AND/OR DETERMINANT AS REQUIRED	315
910	IF (IDET) 1000,527,700	316
527	IF (IVEC) 1000,525,700	317
700	DO 702 I=1,M	318
	DO 702 J=1,M	319
	AR(I,J)=CR(I,J)	320
	IF (I-J) 702,701,702	322
701	AR(I,I)=AR(I,I)-ALR	323
702	CONTINUE	325
	MM=M	326

INTER=0	327
IA=3	
GO TO 535	329
911 IA=4	
IF (IDET) 1000,914,520	331
912 CONTINUE	
ZLAG=AR(1,1)	
ZLIT=ZLAG	335
DO 923 I=2,M	336
ZMAGT=AR(I,I)	
IF (ZLAG-ZMAGT) 922,920,920	338
920 IF (ZLIT-ZMAGT) 923,923,921	339
921 ZLIT=ZMAGT	340
GO TO 923	341
922 ZLAG=ZMAGT	342
923 CONTINUE	343
914 ISL=-1	347
IF (IVEC) 1000,916,915	348
915 DO 703 I=1,M	349
703 XR(I)=0.	
IB=2	
IC=2	
GO TO 530	354
916 IC=3	
GO TO 92	356
704 CONTINUE	
IAARD=IAARD-1	
DO 7051 I=1,M	
7051 VALUR(I,IAARD)=XR(I)	
C	
C FOR WILKINS	
C	
IF(M-IAARD-IQUIT+1)20064,20063,20063	
20063 IF(M-IQUIT)990,20064,990	
20064 CONTINUE	
C	

IB=3	
525 IF (N-1) 526,67,523	360
67 ALR=AR(1,1)	361
SUMR=SUMR+ALR	363
T1=PRDR*ALR	
PRDR=T1	367
ZR(1)=ALR	367A
WRITE(3,320) ALR	
320 FORMAT (20H FINAL EIGENVALUE= E18.8)	
N=0	370
GO TO 910	371
526 CONTINUE	
990 CONTINUE	375
RETURN	
END	376

```

SUBROUTINE WRITE2 (NN,NTAGBC,EX,COMEGA,EVAL,FACTOR,IQUIT)
DOUBLE PRECISION A(42,42),EVEC(42,42),EVAL(42)
DOUBLE PRECISION EX,COMEGA,PI,OMEGA,FACTOR
COMMON EVEC
PI=3.141592653589793

```

C  
C  
C

```

WRITE THE FREQUENCIES AND MODE SHAPES

GO TO (1,2),NTAGBC
1 JJ=NN
  KK=1
  GO TO 3
2 JJ=1
  KK=JJ
3 CONTINUE
  IF(KK-IQUIT)151,151,171
151 CONTINUE
  EVAL(JJ)=FACTOR*EVAL(JJ)

```

```

      GO TO (152,153),NTAGBC
152 EVAL(JJ)=1.00/EVAL(JJ)
153 CONTINUE
      IF(EVAL(JJ))160,162,162
160 WRITE(3,161)KK
161 FORMAT('0',48X,'EIGENVALUE ('',I2,') IS NEGATIVE')
162 EVAL(JJ)=DSQRT(DABS(EVAL(JJ)))
      WRITE(3,163)KK,EVAL(JJ)
163 FORMAT('0',39X,'FREQUENCY ('',I2,') = ',D17.8,' CPS.')
      OMEGA=2.00*PI*COMEGA*EVAL(JJ)
      WRITE(3,10) OMEGA
      10 FORMAT('0',48X,'OMEGA = ',D17.8)
      WRITE(3,164)
164 FORMAT('0',54X,'MODE SHAPE')
      I1=1
      I7=6
      ITAG=1
165 WRITE (3,166) (EVEC(I,JJ),I=I1,I7)
166 FORMAT('0',5X,6D16.8)
      GO TO (167,170),ITAG
167 I1=I1+6
      IF(I7+6-NN)168,169,169
168 I7=I7+6
      GO TO 165
169 I7=NN
      ITAG=2
      GO TO 165
170 GO TO (181,182),NTAGBC
181 JJ=JJ-1
      KK=KK+1
      GO TO 3
182 KK=KK+1
      JJ=KK
      GO TO 3
171 CONTINUE
      RETURN

```

END

```

SUBROUTINE SICI(SI,CI,X)
C
C TEST ARGUMENT RANGE
C
Z=ABS(X)
IF(Z-4.) 10,10,50
C
C Z IS NOT GREATER THAN 4
C
10 Y=Z*Z
OSI=-1.5707963+X*(((.97942154E-11*Y-.22232633E-8)*Y+.30561233E-6
1)*Y-.28341460E-4)*Y+.16666582E-2)*Y-.55555547E-1)*Y+1.)
C
C TEST FOR LOGARITHMIC SINGULARITY
C
IF(Z) 30,20,30
20 CI=-1.E75
RETURN
300CI=0.57721566+ALOG(Z)-Y*(((.13869851E-9*Y+.26945842E-7)*Y-
1.30952207E-5)*Y+.23146303E-3)*Y-.10416642E-1)*Y+.24999999)
40 RETURN
C
C Z IS GREATER THAN 4.
C
50 SI=SIN(Z)
Y=COS(Z)
Z=4./Z
OU=(((.40480690E-2*Z-.022791426)*Z+.055150700)*Z-.072616418)*Z
1+.049877159)*Z-.33325186E-2)*Z-.023146168)*Z-.11349579E-4)*Z
2+.062500111)*Z+.25839886E-9
OY=(((.0051086993*Z+.028191786)*Z-.065372834)*Z+.079020335)*Z
1Z-.044004155)*Z-.0079455563)*Z+.026012930)*Z-.37640003E-3)*Z
```

SICI 038  
SICI 039  
SICI 040  
SICI 041  
SICI 042  
SICI 043  
SICI 044  
SICI 045  
SICI 046  
SICI 047  
SICI 048  
SICI 049  
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SICI 065  
SICI 066  
SICI 067  
SICI 068  
SICI 069

2-.031224178)\*Z-.66464406E-6)\*Z+.25000000

CI=Z\*(SI\*V-Y\*U)

SI=-Z\*(SI\*U+Y\*V)

C  
C  
C

TEST FOR NEGATIVE ARGUMENT

IF(X) 60,40,40

C  
C  
C

X IS LESS THAN -4.

60 SI=-3.1415927-SI

RETURN

END

C

SUBROUTINE QTFE (H,Y,Z,NDIM)

FOR WRITE-UP, SEE IBM SSP .

DIMENSION Y(1),Z(1)

SUM2=0.

IF(NDIM-1)4,3,1

1 HH=.5\*H

DO 2 I=2,NDIM

SUM1=SUM2

SUM2=SUM2+HH\*(Y(I)+Y(I-1))

2 Z(I-1)=SUM1

3 Z(NDIM)=SUM2

4 RETURN

END

SICI 070

SICI 071

SICI 072

SICI 073

SICI 074

SICI 075

SICI 076

SICI 077

SICI 078

SICI 079

SICI 080

SICI 081

SICI 082

```

C.  DOUBLE PRECISION FUNCTION IR111 (K,M)
    FREELY SUPPORTED
    DIMENSION FUNC(101),ANS(101)
    DOUBLE PRECISION E
    DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
    1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
    2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
    3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
    4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
    5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
    6FREE55(6,6)
    COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
    1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
    2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
    3FE55,FREE55,H,XMU,NT(7)
    N=50
    NDIM=N+1
    P=-2.*(1.+XMU)
    IF(SA)4,1,4
    1 IF(M-K)3,2,3
    2 IR111=.5D0/(ROB**(2.+2.*XMU))
    GO TO 100
    3 IR111=0.D0
    GO TO 100
    4 E=0.
    I=1
    DELTA=1./FLOAT(N)
    5 FUNC(I)=(ROB+E*SA)**P * DCOS(DFLOAT(M)*E*PI)*DCOS(DFLOAT(K)*E*PI)
    IF(I-N)6,6,7
    6 I=I+1
    E=E+DELTA
    GO TO 5
    7 CALL QTFE (DELTA,FUNC,ANS,NDIM)
    IR111 = ANS(NDIM)
100 RETURN
    END

```

```

C:  DOUBLE PRECISION FUNCTION IR11 (K,M)
    FREELY SUPPORTED
    DIMENSION FUNC(101),ANS(101)
    DOUBLE PRECISION E
    DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
    COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
    N=50
    NDIM=N+1
    P=-2.*XMU
    IF(SA)4,1,4
1 IF(M-K)3,2,3
2 IR11=.5DO*ROB**(-2.*XMU)
    GO TO 100
3 IR11=0.DO
    GO TO 100
4 E=0.
    I=1
    DELTA=1./FLOAT(N)
5 FUNC(I)=(ROB+E*SA)**P * DCOS(DFLOAT(M)*E*PI)*DCOS(DFLOAT(K)*E*PI)
    IF(I-N)6,6,7
6 I=I+1
    E=E+DELTA
    GO TO 5
7 CALL QTFE (DELTA,FUNC,ANS,NDIM)
    IR11 = ANS(NDIM)
100 RETURN
    END

```

```

C      DOUBLE PRECISION FUNCTION IE11 (K,M)
      FREELY SUPPORTED
      DIMENSION FUNC(101),ANS(101)
      DOUBLE PRECISION E
      DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
      COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
      N=50
      NDIM=N+1
      P=-1.-2.*XMU
      IF(SA)4,1,4
1  IF(DFLOAT(M+K)/2.DO-(M+K)/2)3,2,3
2  IE11=0.DO
      GO TO 100
3  IE11=-2.DO*DFLOAT(M*M)*ROB**(-1.-2.*XMU)/DFLOAT(M*M-K*K)
      GO TO 100
4  E=0.
      I=1
      DELTA=1./FLOAT(N)
5  FUNC(I)=(ROB+E*SA)**P * DSIN(DFLOAT(M)*E*PI)*DCOS(DFLOAT(K)*E*PI)
      IF(I-N)6,6,7
6  I=I+1
      E=E+DELTA
      GO TO 5
7  CALL QTFE (DELTA,FUNC,ANS,NDIM)
      IE11 = -XMU*SA*FR111(K,M)-DFLOAT(M)*PI*ANS(NDIM)
100 RETURN
      END

```

```

C
DOUBLE PRECISION FUNCTION IREE11(K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FEE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE11=XMU*SA*(1.-XMU*SA)*FR11(K,M)-DFLOAT(N*M)*PI*PI*FR11(K,M)
1
-2.*XMU*SA*FE11(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IR121 (K,M)
FREELY SUPPORTED
DIMENSION FUNC(101),ANS(101)
DOUBLE PRECISION E
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
N=50
NDIM=N+1
P=-2.-XMU
IF(SA)4,1,4

```

```

1 IF(DFLOAT(M+K)/2.DO-(M+K)/2)3,2,3
2 IR121=0.DO
  GO TO 100
3 IR121=2.DO*DFLOAT(M)*ROB**{-2.-XMU)/(PI*DFLOAT(M*M-K*K))
  GO TO 100
4 E=0.
  I=1
  DELTA=1./FLOAT(N)
5 FUNC(I)=(ROB+E*SA)**P * DSIN(DFLOAT(M)*E*PI)*DCOS(DFLOAT(K)*E*PI)
  IF(I-N)6,6,7
6 I=I+1
  E=E+DELTA
  GO TO 5
7 CALL QTFE (DELTA,FUNC,ANS,NDIM)
  IR121 = ANS(NDIM)
100 RETURN
  END
  DOUBLE PRECISION FUNCTION IE21 (K,M)
  FREELY SUPPORTED
  DIMENSION FUNC(101),ANS(101)
  DOUBLE PRECISION E
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
  COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
  N=50
  NDIM=N+1
  P=-1.-XMU
  IF(SA)4,1,4

```

```

1 IF(M-K)3,2,3
2 IE21=.5D0*DFLOAT(M)*PI*ROB**(-1.-XMU)
  GO TO 100
3 IE21=0.D0
  GO TO 100
4 E=0.
  I=1
  DELTA=1./FLOAT(N)
5 FUNC(I)=(ROB+E*SA)**P * DCOS(DFLOAT(M)*E*PI)*DCOS(DFLOAT(K)*E*PI)
  IF(I-N)6,6,7
6 I=I+1
  E=E+DELTA
  GO TO 5
7 CALL QTFE (DELTA,FUNC,ANS,NDIM)
  IE21 = M*PI*ANS(NDIM)
100 RETURN
  END

```

```

C
  DOUBLE PRECISION FUNCTION IR131 (K,M)
  FREELY SUPPORTED
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
  COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
  IR131=FR121(K,M)
  RETURN
  END

```

```

C
  DOUBLE PRECISION FUNCTION IE31 (K,M)
  FREELY SUPPORTED
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,

```

```

1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE31=FE21(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IE12 (K,M)
FREELY SUPPORTED
DIMENSION FUN1(101),FUN2(101),ANS1(101),ANS2(101)
DOUBLE PRECISION E
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
N=50
NDIM=N+1
PI=-2.-XMU
P2=-1.-XMU
IF(SA)4,1,4
1 IF(M-K)3,2,3
2 IE12=-.5D0*PI*DFLOAT(M)*ROB**(-1.-XMU)
GO TO 100

```

C

```

3 IE12=0.00
GO TO 100
4 E=0.
I=1
DELTA=1./FLOAT(N)
5 FUN1(I)=(ROB+E*SA)**PI* DCOS(DFLOAT(M)*E*PI)*DSIN(DFLOAT(K)*E*PI)
FUN2(I)=(ROB+E*SA)**P2* DSIN(DFLOAT(M)*E*PI)*DSIN(DFLOAT(K)*E*PI)
IF(I-N)6,6,7
6 I=I+1
E=E+DELTA
GO TO 5
7 CALL QTFE (DELTA,FUN1,ANS1,NDIM)
CALL QTFE (DELTA,FUN2,ANS2,NDIM)
IE12 = -XMU*SA*ANS1(NDIM)-M*PI*ANS2(NDIM)
100 RETURN
END

```

C

```

DOUBLE PRECISION FUNCTION IR122 (K,M)
FREELY SUPPORTED
DOUBLE PRECISION DS,DD
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(SA)6,1,6
1 IF(M-K)5,2,5
2 IF(M)4,3,4
3 IR122=0.00
GO TO 18
4 IR122=.500/(ROB*ROB)

```

```

GO TO 18
5 IR122=0.DO
GO TO 18
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
RHOBAR=ROB/SA
XSI=DS*RHOBAR
XSF=DS*(RHOBAR+1.)
CS=COS(XSI)
SS=SIN(XSI)
CALL SICI (SSI,CSI,XSI)
CALL SICI (SSF,CSF,XSF)
T1=.5*DS*(CS*(SSF-SSI)-SS*(CSF-CSI))/(SA*SA)
IF(M-K)8,7,8
7 T2=0.
GO TO 17
8 XDI=DD*RHOBAR
XDF=DD*(RHOBAR+1.)
CD=COS(XDI)
SD=SIN(XDI)
CALL SICI (SDI,CDI,XDI)
CALL SICI (SDF,CDF,XDF)
T2=-.5*DD*(CD*(SDF-SDI)-SD*(CDF-CDI))/(SA*SA)
17 IR122=T1+T2
18 RETURN
END
DOUBLE PRECISION FUNCTION IR22 (K,M)
C FREELY SUPPORTED
IF(M-K)2,1,2
1 IR22=.5D0
GO TO 3
2 IR22=0.DO
3 RETURN
END
DOUBLE PRECISION FUNCTION IE22 (K,M)
C FREELY SUPPORTED

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```

DOUBLE PRECISION DS,DD
DOUBLE PRECISION FR11(6,6),FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(SA)6,1,6
1 IF(DFLOAT(M+K)/2.0D0-(M+K)/2)2,3,2
2 IE22=+2.0D0*DFLOAT(K*M)/(ROB*DFLOAT(K*K-M*M))
GO TO 18
3 IE22=0.0D0
GO TO 18
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
RHOBAR=ROB/SA
XSI=DS*RHOBAR
XSF=DS*(RHOBAR+1.)
CS=COS(XSI)
SS=SIN(XSI)
CALL SICI (SSI,CSI,XSI)
CALL SICI (SSF,CSF,XSF)
T1=.5D0*PI*DFLOAT(M)/SA*(CS*(SSF-SSI)-SS*(CSF-CSI))
IF(M-K)8,7,8
7 T2=0.
GO TO 17
8 XDI=DD*RHOBAR
XDF=DD*(RHOBAR+1.)
CD=COS(XDI)
SD=SIN(XDI)
CALL SICI (SDI,CDI,XDI)

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```

CALL SICI (SDF,CDF,XDF)
T2=-.5D0*PI*DFLOAT(M)/SA*(CD*(SDF-SDI)-SD*(CDF-CDI))
17 IE22=T1+T2
18 RETURN

```

```

END

```

```

DOUBLE PRECISION FUNCTION IREE22(K,M)

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```

FREELY SUPPORTED

```

```

DOUBLE PRECISION FR11(6,6),FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)

```

```

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)

```

```

IREE22=-M*M*PI*PI*FR22(K,M)

```

```

RETURN

```

```

END

```

```

DOUBLE PRECISION FUNCTION IR132 (K,M)

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```

FREELY SUPPORTED

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```

DOUBLE PRECISION FR11(6,6),FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)

```

```

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)

```

```

IR132=FR122(K,M)

```

```

RETURN

```

```

C

```

END

DOUBLE PRECISION FUNCTION I42 (K,M)

C FREELY SUPPORTED

DOUBLE PRECISION RHOBAR,CSR,SSR,CDR,SDR,DS,DD

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

C  
C  
C

FOR A CYLINDER

IF(SA)6,1,6  
1 IF(M-K)5,2,5  
2 IF(M)4,3,4  
3 I42=1.D0/ROB  
GO TO 18  
4 I42=.5D0/ROB  
GO TO 18  
5 I42=0.D0  
GO TO 18

C  
C  
C

FOR A CONE

6 DS=DFLOAT(M+K)  
DD=DFLOAT(M-K)  
IF(M-K)7,8,7  
7 ITAG=2  
GO TO 11  
8 IF(M)10,9,10

```

9 I42=0.D0
  GO TO 18
10 ITAG=1
11 RHOBAR=ROB/SA
  XSI=DS*PI*RHOBAR
  XSF=DS*PI*(RHOBAR+1.D0)
  CSR=COS(XSI)
  SSR=SIN(XSI)
  CALL SICI (SXSI,CXSI,XSI)
  CALL SICI (SXSF,CXSF,XSF)
  GOTO (14,15),ITAG
14 I42=.5D0*(-CSR*(CXSF-CXSI)-SSR*(SXSF-SXSI)+DLOG(1.D0+SA/ROB))/SA
  GO TO 18
15 XDI=DD*PI*RHOBAR
  XDF=DD*PI*(RHOBAR+1.D0)
  CDR=COS(XDI)
  SDR=SIN(XDI)
  CALL SICI (SXDI,CXDI,XDI)
  CALL SICI (SXDF,CXDF,XDF)
  I42=.5D0*(-CSR*(CXSF-CXSI)-SSR*(SXSF-SXSI)
1   +CDR*(CXDF-CXDI)+SDR*(SXDF-SXDI))/SA
18 RETURN
  END
  DOUBLE PRECISION FUNCTION IE13 (K,M)
  FREELY SUPPORTED
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
  COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)

```

IE13=FE12(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR133 (K,M)

C

FREELY SUPPORTED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IR133=FR122(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR33 (K,M)

C

FREELY SUPPORTED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IR33=FR22(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE33 (K,M)

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C
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE33=FE22(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IREE33(K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE33=FREE22(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION I43(K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,

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3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
I43=F42(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION I53 (K,M)
FREELY SUPPORTED
DIMENSION FUNC(101),ANS(101)
DOUBLE PRECISION E
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
N=50
NDIM=N+1
P=-1.-XMU
IF(SA)4,1,4
1 IF(DFLOAT(M+K)/2.DO-(M+K)/2)3,2,3
2 I53=0.DO
GO TO 100
3 I53=2.DO*DFLOAT(K)*ROB**(-1.-XMU)/(PI*DFLOAT(K*K-M*M))
GO TO 100
4 E=0.

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I=1
DELTA=1./FLOAT(N)
5 FUNC(I)=(ROB+E*SA)**P * DCOS(DFLOAT(M)*E*PI)*DSIN(DFLOAT(K)*E*PI)
IF(I-N)6,6,7
6 I=I+1
E=E+DELTA
GO TO 5
7 CALL QTFE (DELTA,FUNC,ANS,NDIM)
I53 = ANS(NDIM)
100 RETURN
END
DOUBLE PRECISION FUNCTION IRE53 (K,M)
FREELY SUPPORTED
DIMENSION FUN1(101),FUN2(101),ANS1(101),ANS2(101)
DOUBLE PRECISION E
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
N=50
NDIM=N+1
P1=-1.-XMU
P2=-XMU
IF(SA)4,1,4
1 IF(M-K)3,2,3
2 IRE53=-.5D0*DFLOAT(N)*PI*ROB**(-XMU)
GO TO 100
3 IRE53=0.D0
GO TO 100

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4 E=0.
  I=1
  DELTA=1./FLOAT(N)
5 FUN1(I)=(ROB+E*SA)**P1* DCOS(DFLOAT(M)*E*PI)*DSIN(DFLOAT(K)*E*PI)
  FUN2(I)=(ROB+E*SA)**P2* DSIN(DFLOAT(M)*E*PI)*DSIN(DFLOAT(K)*E*PI)
  IF(I-N)6,6,7
6 I=I+1
  E=E+DELTA
  GO TO 5
7 CALL QTFE (DELTA,FUN1,ANS1,NDIM)
  CALL QTFE (DELTA,FUN2,ANS2,NDIM)
  IRE53 = -XMU*SA*ANS1(NDIM)-M*PI*ANS2(NDIM)
100 RETURN
  END
DOUBLE PRECISION FUNCTION IR144 (K,M)
C FREELY SUPPORTED
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
  COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
  IR144=FR122(K,M)
  RETURN
  END
DOUBLE PRECISION FUNCTION IR44 (K,M)
C FREELY SUPPORTED
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,

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4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR44=FR22(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IE44 (K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE44=FE22(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IREE44(K,M)
FREELY SUPPORTED
IREE44=FREE22(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IR154 (K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,

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3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR154=FR121(M,K)
RETURN
END
C
DOUBLE PRECISION FUNCTION IE54 (K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE54=FE12(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IRE35 (K,M)
FREELY SUPPORTED
DIMENSION FUNC(101),ANS(101)
DOUBLE PRECISION E
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),

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5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
N=50
NDIM=N+1
P=-XMU
IF(SA)4,1,4
1 IF(M-K)3,2,3
2 IRE35=-.500*DFLOAT(M)*PI*ROB**(-XMU)
GO TO 100
3 IRE35=0.00
GO TO 100
4 E=0.
I=1
DELTA=1./FLOAT(N)
5 FUNC(I)={ROB+E*SA}**P * DCOS(DFLOAT(M)*E*PI)*DCOS(DFLOAT(K)*E*PI)
IF(I-N)6,6,7
6 I=I+1
E=E+DELTA
GO TO 5
7 CALL QTFE (DELTA,FUNC,ANS,NDIM)
IRE35= M*PI*ANS(NDIM)
100 RETURN
END
DOUBLE PRECISION FUNCTION IE45 (K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)

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COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IE45=FE21(K,M)  
 RETURN

END

DOUBLE PRECISION FUNCTION IR155 (K,M)

FREELY SUPPORTED

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR155=FR11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR55 (K,M)

FREELY SUPPORTED

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)

C

C

```

IR55=FR11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IE55 (K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE55=FE11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IREE55(K,M)
FREELY SUPPORTED
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE55=FREE11(K,M)
RETURN
END
SUBROUTINE QTFE (H,Y,Z,NDIM)

```

C

C

C . FOR WRITE-UP, SEE IBM SSP .  
DIMENSION Y(1),Z(1)  
SUM2=0.  
IF(NDIM-1)4,3,1  
1 HH=.5\*H  
DO 2 I=2,NDIM  
SUM1=SUM2  
SUM2=SUM2+HH\*(Y(I)+Y(I-1))  
2 Z(I-1)=SUM1  
3 Z(NDIM)=SUM2  
4 RETURN  
END

```

C
DOUBLE PRECISION FUNCTION IR111 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION DS,DD
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(SA)6,1,6
1 IF(M-K)5,2,5
2 IF(M)4,3,4
3 IR111=0.0D0
GO TO 18
4 IR111=.5D0/(ROB*ROB)
GO TO 18
5 IR111=0.0D0
GO TO 18
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
RHOBAR=ROB/SA
XSI=DS*RHOBAR
XSF=DS*(RHOBAR+1.)
CS=COS(XSI)
SS=SIN(XSI)
CALL SICI (SSI,CSI,XSI)
CALL SICI (SSF,CSF,XSF)
T1=.5*DS*(CS*(SSF-SSI)-SS*(CSF-CSI))/(SA*SA)
IF(M-K)8,7,8
7 T2=0.
GO TO 17

```

```

8 XDI=DD*RHOBAR
  XDF=DD*(RHOBAR+1.)
  CD=COS(XDI)
  SD=SIN(XDI)
  CALL SICI (SDI,CDI,XDI)
  CALL SICI (SDF,CDF,XDF)
  T2=-.5*DD*(CD*(SDF-SDI)-SD*(CDF-CDI))/(SA*SA)
17 IR11=T1+T2
18 RETURN
  END

```

```

C DOUBLE PRECISION FUNCTION IR11 (K,M)
  CLAMPED - CLAMPED
  IF(M-K)4,1,4
1 IF(M)3,2,3
2 IR11=1.00
  GO TO 5
3 IR11=.500
  GO TO 5
4 IR11=0.00
5 RETURN
  END

```

```

C DOUBLE PRECISION FUNCTION IE11 (K,M)
  CLAMPED - CLAMPED
  DOUBLE PRECISION DS,DD
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
  COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
  IF(SA)6,1,6

```

```

1 IF(DFLOAT(M+K)/2.D0-(M+K)/2)2,3,2
2 IE11=+2.D0*DFLOAT(K*M)/(ROB*DFLOAT(K*K-M*M))
GO TO 18
3 IE11=0.D0
GO TO 18
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
RHOBAR=ROB/SA
XSI=DS*RHOBAR
XSF=DS*(RHOBAR+1.)
CS=COS(XSI)
SS=SIN(XSI)
CALL SICI (SSI,CSI,XSI)
CALL SICI (SSF,CSF,XSF)
T1=.5D0*PI*DFLOAT(M)/SA*(CS*(SSF-SSI)-SS*(CSF-CSI))
IF(M-K)8,7,8
7 T2=0.
GO TO 17
8 XDI=DD*RHOBAR
XDF=DD*(RHOBAR+1.)
CD=COS(XDI)
SD=SIN(XDI)
CALL SICI (SDI,CDI,XDI)
CALL SICI (SDF,CDF,XDF)
T2=-.5D0*PI*DFLOAT(M)/SA*(CD*(SDF-SDI)-SD*(CDF-CDI))
17 IE11=T1+T2
18 RETURN
END
DOUBLE PRECISION FUNCTION IREE11(K,M)
CLAMPED - CLAMPED
C DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,

```

```

6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
1REE11=-M*M*PI*PI*FR11(K,M)
RETURN
END

```

```

C
DOUBLE PRECISION FUNCTION IR121 (K,M)
CLAMPED - CLAMPED

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```

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)

```

```

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR121=FR111(K,M)
RETURN
END

```

```

C
DOUBLE PRECISION FUNCTION IE21 (K,M)
CLAMPED - CLAMPED

```

```

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)

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```

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,

```

3FE55, FREE55, H, XMU, NT(7)

IE21=FE11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR131 (K, M)

CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6), FR11(6,6), FE11(6,6), FREE11(6,6), ROB,  
 1FR121(6,6), FE21(6,6), FR131(6,6), FE31(6,6), FE12(6,6), FR122(6,6),  
 2FR22(6,6), FE22(6,6), FREE22(6,6), FR132(6,6), F42(6,6), FE13(6,6), H,  
 3FR133(6,6), FR33(6,6), FE33(6,6), FREE33(6,6), F43(6,6), F53(6,6), SA,  
 4FRE53(6,6), FR144(6,6), FR44(6,6), FE44(6,6), FREE44(6,6), FR154(6,6),  
 5FE54(6,6), FRE35(6,6), FE45(6,6), FR155(6,6), FR55(6,6), FE55(6,6), PI,  
 6FREE55(6,6)

COMMON FR111, FR11, FE11, FREE11, FR121, FE21, FR131, FE31, FE12, FR122, SA,  
 1FR22, FE22, FREE22, FR132, F42, FE13, FR133, FR33, FE33, FREE33, F43, F53, PI,  
 2FRE53, FR144, FR44, FE44, FREE44, FR154, FE54, FRE35, FE45, FR155, FR55, ROB,  
 3FE55, FREE55, H, XMU, NT(7)

IR131=FR111(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE31 (K, M)

CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6), FR11(6,6), FE11(6,6), FREE11(6,6), ROB,  
 1FR121(6,6), FE21(6,6), FR131(6,6), FE31(6,6), FE12(6,6), FR122(6,6),  
 2FR22(6,6), FE22(6,6), FREE22(6,6), FR132(6,6), F42(6,6), FE13(6,6), H,  
 3FR133(6,6), FR33(6,6), FE33(6,6), FREE33(6,6), F43(6,6), F53(6,6), SA,  
 4FRE53(6,6), FR144(6,6), FR44(6,6), FE44(6,6), FREE44(6,6), FR154(6,6),  
 5FE54(6,6), FRE35(6,6), FE45(6,6), FR155(6,6), FR55(6,6), FE55(6,6), PI,  
 6FREE55(6,6)

COMMON FR111, FR11, FE11, FREE11, FR121, FE21, FR131, FE31, FE12, FR122, SA,  
 1FR22, FE22, FREE22, FR132, F42, FE13, FR133, FR33, FE33, FREE33, F43, F53, PI,  
 2FRE53, FR144, FR44, FE44, FREE44, FR154, FE54, FRE35, FE45, FR155, FR55, ROB,  
 3FE55, FREE55, H, XMU, NT(7)

IE31=FE11(K,M)

RETURN

END

C

C

DOUBLE PRECISION FUNCTION IE12 (K,M)  
 C CLAMPED - CLAMPED  
 DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)  
 COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IE12=FE11(K,M)  
 RETURN  
 END

C DOUBLE PRECISION FUNCTION IR122 (K,M)  
 CLAMPED - CLAMPED  
 DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)  
 COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR122=FR111(K,M)  
 RETURN  
 END

C DOUBLE PRECISION FUNCTION IR22 (K,M)  
 CLAMPED - CLAMPED  
 DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),

2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IR22=FR11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE22 (K,M)

C CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IE22=FE11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IREE22(K,M)

C CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,

6FREE55(6,6)  
 COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IREE22=FREE11(K,M)  
 RETURN  
 END

END

DOUBLE PRECISION FUNCTION IR132 (K,M)

C CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR132=FR111(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION I42 (K,M)

C CLAMPED - CLAMPED

DOUBLE PRECISION I53

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,

2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 I42=I53(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE13 (K,M)

C CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IE13=FE11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR133 (K,M)

C CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)

IR133=FR111(K,M)

RETURN

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END
DOUBLE PRECISION FUNCTION IR33 (K,M)
C CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR33=FR11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IE33 (K,M)
C CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE33=FE11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IREE33(K,M)
C CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,

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1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FEE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
FREE33=FREE11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION I43 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FEE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
I43=F42(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION I53 (K,M)
DOUBLE PRECISION RHOBAR,CSR,SSR,CDR,SDR,DS,DD
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),

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C

5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)  
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

C  
C  
C

FOR A CYLINDER

IF(SA)6,1,6  
1 IF(M-K)5,2,5  
2 IF(M)4,3,4  
3 I53 =1.00/ROB  
GO TO 18  
4 I53 =.500/ROB  
GO TO 18  
5 I53 =0.00  
GO TO 18

C  
C  
C

FOR A CONE

6 DS=DFLOAT(M+K)  
DD=DFLOAT(M-K)  
IF(M-K)7,8,7  
7 ITAG=2  
GO TO 11  
8 IF(M)10,9,10  
9 I53=0.00  
GO TO 18  
10 ITAG=1  
11 RHOBAR=ROB/SA  
XSI=DS\*PI\*RHOBAR  
XSF=DS\*PI\*(RHOBAR+1.00)  
CSR=COS(XSI)  
SSR=SIN(XSI)  
CALL SICI (SXSI,CXSI,XSI)

```

CALL SICI (SXSF,CXSF,XSF)
GOTO (14,15),ITAG
14 I53=.5D0*(-CSR*(CXSF-CXSI))-SSR*(SXSF-SXSI)+DLOG(1.D0+SA/ROB))/SA
GO TO 18
15 XDI=DD*PI*RHOBAR
XDF=DD*PI*(RHOBAR+1.D0)
CDR=COS(XDI)
SDR=SIN(XDI)
CALL SICI (SXDI,CXDI,XDI)
CALL SICI (SXDF,CXDF,XDF)
I53 =.5D0*(-CSR*(CXSF-CXSI))-SSR*(SXSF-SXSI)
1 +CDR*(CXDF-CXDI)+SDR*(SXDF-SXDI))/SA
18 RETURN
END
C
DOUBLE PRECISION FUNCTION IRE53 (K,M)
CLAMPED = CLAMP
DOUBLE PRECISION DM2,DK2
DOUBLE PRECISION FR11(6,6),FR12(6,6),FR13(6,6),FR14(6,6),FR15(6,6),FR16(6,6),FR17(6,6),FR18(6,6),FR19(6,6),FR20(6,6),FR21(6,6),FR22(6,6),FR23(6,6),FR24(6,6),FR25(6,6),FR26(6,6),FR27(6,6),FR28(6,6),FR29(6,6),FR30(6,6),FR31(6,6),FR32(6,6),FR33(6,6),FR34(6,6),FR35(6,6),FR36(6,6),FR37(6,6),FR38(6,6),FR39(6,6),FR40(6,6),FR41(6,6),FR42(6,6),FR43(6,6),FR44(6,6),FR45(6,6),FR46(6,6),FR47(6,6),FR48(6,6),FR49(6,6),FR50(6,6),FR51(6,6),FR52(6,6),FR53(6,6),FR54(6,6),FR55(6,6),FR56(6,6),FR57(6,6),FR58(6,6),FR59(6,6),FR60(6,6),FR61(6,6),FR62(6,6),FR63(6,6),FR64(6,6),FR65(6,6),FR66(6,6),FR67(6,6),FR68(6,6),FR69(6,6),FR70(6,6),FR71(6,6),FR72(6,6),FR73(6,6),FR74(6,6),FR75(6,6),FR76(6,6),FR77(6,6),FR78(6,6),FR79(6,6),FR80(6,6),FR81(6,6),FR82(6,6),FR83(6,6),FR84(6,6),FR85(6,6),FR86(6,6),FR87(6,6),FR88(6,6),FR89(6,6),FR90(6,6),FR91(6,6),FR92(6,6),FR93(6,6),FR94(6,6),FR95(6,6),FR96(6,6),FR97(6,6),FR98(6,6),FR99(6,6),FR100(6,6)
DOUBLE PRECISION FE11(6,6),FE12(6,6),FE13(6,6),FE14(6,6),FE15(6,6),FE16(6,6),FE17(6,6),FE18(6,6),FE19(6,6),FE20(6,6),FE21(6,6),FE22(6,6),FE23(6,6),FE24(6,6),FE25(6,6),FE26(6,6),FE27(6,6),FE28(6,6),FE29(6,6),FE30(6,6),FE31(6,6),FE32(6,6),FE33(6,6),FE34(6,6),FE35(6,6),FE36(6,6),FE37(6,6),FE38(6,6),FE39(6,6),FE40(6,6),FE41(6,6),FE42(6,6),FE43(6,6),FE44(6,6),FE45(6,6),FE46(6,6),FE47(6,6),FE48(6,6),FE49(6,6),FE50(6,6),FE51(6,6),FE52(6,6),FE53(6,6),FE54(6,6),FE55(6,6),FE56(6,6),FE57(6,6),FE58(6,6),FE59(6,6),FE60(6,6),FE61(6,6),FE62(6,6),FE63(6,6),FE64(6,6),FE65(6,6),FE66(6,6),FE67(6,6),FE68(6,6),FE69(6,6),FE70(6,6),FE71(6,6),FE72(6,6),FE73(6,6),FE74(6,6),FE75(6,6),FE76(6,6),FE77(6,6),FE78(6,6),FE79(6,6),FE80(6,6),FE81(6,6),FE82(6,6),FE83(6,6),FE84(6,6),FE85(6,6),FE86(6,6),FE87(6,6),FE88(6,6),FE89(6,6),FE90(6,6),FE91(6,6),FE92(6,6),FE93(6,6),FE94(6,6),FE95(6,6),FE96(6,6),FE97(6,6),FE98(6,6),FE99(6,6),FE100(6,6)
COMMON FR11,FR12,FR13,FR14,FR15,FR16,FR17,FR18,FR19,FR20,FR21,FR22,FR23,FR24,FR25,FR26,FR27,FR28,FR29,FR30,FR31,FR32,FR33,FR34,FR35,FR36,FR37,FR38,FR39,FR40,FR41,FR42,FR43,FR44,FR45,FR46,FR47,FR48,FR49,FR50,FR51,FR52,FR53,FR54,FR55,FR56,FR57,FR58,FR59,FR60,FR61,FR62,FR63,FR64,FR65,FR66,FR67,FR68,FR69,FR70,FR71,FR72,FR73,FR74,FR75,FR76,FR77,FR78,FR79,FR80,FR81,FR82,FR83,FR84,FR85,FR86,FR87,FR88,FR89,FR90,FR91,FR92,FR93,FR94,FR95,FR96,FR97,FR98,FR99,FR100
COMMON FE11,FE12,FE13,FE14,FE15,FE16,FE17,FE18,FE19,FE20,FE21,FE22,FE23,FE24,FE25,FE26,FE27,FE28,FE29,FE30,FE31,FE32,FE33,FE34,FE35,FE36,FE37,FE38,FE39,FE40,FE41,FE42,FE43,FE44,FE45,FE46,FE47,FE48,FE49,FE50,FE51,FE52,FE53,FE54,FE55,FE56,FE57,FE58,FE59,FE60,FE61,FE62,FE63,FE64,FE65,FE66,FE67,FE68,FE69,FE70,FE71,FE72,FE73,FE74,FE75,FE76,FE77,FE78,FE79,FE80,FE81,FE82,FE83,FE84,FE85,FE86,FE87,FE88,FE89,FE90,FE91,FE92,FE93,FE94,FE95,FE96,FE97,FE98,FE99,FE100
3FE55,FREE55,H,XMU,NT(7)
1 IF(DFLOAT(M+K)/2.D0-(M+K)/2)1,3,1
1 IF(M)2,3,2
2 DM2=DFLOAT(M)*DFLOAT(M)
DK2=DFLOAT(K)*DFLOAT(K)
IRE53=+2.D0*K*M/(DK2-DM2)
GO TO 4
3 IRE53=0.D0
4 RETURN

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END
DOUBLE PRECISION FUNCTION IR144 (K,M)
C   CLAMPED - CLAMPED
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR144=FR111(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IR44 (K,M)
C   CLAMPED - CLAMPED
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR44=FR11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IE44 (K,M)
C   CLAMPED - CLAMPED
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,

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1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FEE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE44=FE11(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IREE44(K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FEE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE44=FREE11(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IR154 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),

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5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR154=FR111(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IE54 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE54=FE11(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IRE35 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION IRE35
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,

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1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IRE35=IRE53(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION IE45 (K,M)

CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IE45=FE11(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION IR155 (K,M)

CLAMPED - CLAMPED

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR155=FR111(K,M)

```

RETURN
END
DOUBLE PRECISION FUNCTION IR55 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR122(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR55=FR11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IE55 (K,M)
CLAMPED - CLAMPED
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE55=FE11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IREE55(K,M)
CLAMPED - CLAMPED

```

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IREE55=FREE11(K,M)

RETURN

END

C

```
DOUBLE PRECISION FUNCTION IR111 (K,M)
FREE - FREE
DOUBLE PRECISION DS,DD,DM,DK
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(SA)6,1,6
1 IF(M-K)5,2,5
2 IF(M)4,3,4
3 IR111=1.00/(ROB*ROB)
GO TO 18
4 IR111=.500/(ROB*ROB)
GO TO 18
5 IR111=0.00
GO TO 18
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
IF(M-K)12,10,12
10 IF(M)12,11,12
11 IR111=(1.00/ROB-1.00/(ROB+SA))/SA
GO TO 18
12 CONTINUE
RHOBAR=ROB/SA
XSI=DS*RHOBAR
XSF=DS*(RHOBAR+1.)
CS=COS(XSI)
SS=SIN(XSI)
CALL SICI (SSI,CSI,XSI)
```

```

CALL SICI (SSF,CSF,XSF)
TO=-((( -1)**(M+K))/(ROB+SA)-1.DO/ROB)/SA
T1=-.5*DS/(SA*SA)*(CS*(SSF-SSI)-SS*(CSF-CSI))
IF(M-K)8,7,8
7 T2=0.
GO TO 17
8 XDI=DD*RHOBAR
XDF=DD*(RHOBAR+L.)
CD=COS(XDI)
SD=SIN(XDI)
CALL SICI (SDI,CDI,XDI)
CALL SICI (SDF,CDF,XDF)
T2=-.5*DD/(SA*SA)*(CD*(SDF-SDI)-SD*(CDF-CDI))
17 IR11=TO+T1+T2
18 RETURN
END

```

```

C
DOUBLE PRECISION FUNCTION IR11 (K,M)
FREE - FREE
IF(M-K)4,1,4
1 IF(M)3,2,3
2 IR11=1.DO
GO TO 5
3 IR11=.5DO
GO TO 5
4 IR11=0.DO
5 RETURN
END

```

```

C
DOUBLE PRECISION FUNCTION IE11 (K,M)
FREE - FREE
DOUBLE PRECISION DS,DD
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,

```

```

6FREE5(6,6)
COMMON FR11,FR11,FE11,FR121,FE21,FR131,FE31,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
  IF(M)2,1,2
1 IE11=0.00
  GO TO 18
2 IF(SA)6,3,6
3 IF(DFLOAT(M+K)/2.00-(M+K)/2)4,1,4
4 IE11=-2.*DFLOAT(M*N)/(ROB*DFLOAT(M*M-K*K))
  GO TO 18
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
RHOBAR=ROB/SA
XSI=DS*RHOBAR
XSF=DS*(RHOBAR+1.)
CS=COS(XSI)
SS=SIN(XSI)
CALL SICI (SSI,CSI,XSI)
CALL SICI (SSF,CSF,XSF)
T1=-.5*PI*DFLOAT(M)/SA*(CS*(SSF-SSI)-SS*(CSF-CSI))
IF(M-K)8,7,8
7 T2=0.
  GO TO 17
8 XDI=DD*RHOBAR
XDF=DD*(RHOBAR+1.)
CD=COS(XDI)
SD=SIN(XDI)
CALL SICI (SDI,CDI,XDI)
CALL SICI (SDF,CDF,XDF)
T2=-.5*PI*DFLOAT(M)/SA*(CD*(SDF-SDI)-SD*(CDF-CDI))
17 IE11=T1+T2
18 RETURN
END
DOUBLE PRECISION FUNCTION IREE11(K,M)

```

```

C
FREE - FREE
DOUBLE PRECISION IR11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE11=-M*M*PI*PI*IR11(K,N)
RETURN
END

C
DOUBLE PRECISION FUNCTION IR121 (K,M)
FREE - FREE
DOUBLE PRECISION IR11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR121=IR11(K,M)
RETURN
END

C
DOUBLE PRECISION FUNCTION IE21 (K,M)
FREE - FREE
DOUBLE PRECISION IE11

```

```
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)
```

```
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)
```

```
IE21=IE11(K,M)
```

```
RETURN
```

```
END
```

```
DOUBLE PRECISION FUNCTION IR131 (K,M)
```

```
FREE - FREE
```

```
DOUBLE PRECISION IR111
```

```
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)
```

```
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)
```

```
IR131=IR111(K,M)
```

```
RETURN
```

```
END
```

```
DOUBLE PRECISION FUNCTION IE31 (K,M)
```

```
FREE - FREE
```

```
DOUBLE PRECISION IE11
```

```
DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
```

2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)  
 COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IE31=IE11(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION IE12 (K,M)

C

FREE - FREE

DOUBLE PRECISION IE11

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IE12=IE11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR122 (K,M)

FREE - FREE

DOUBLE PRECISION IR111

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,

C

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4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR122=IR11(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IR22 (K,M)
FREE - FREE
DOUBLE PRECISION IR11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IR22=IR11(K,M)
RETURN
END
C
DOUBLE PRECISION FUNCTION IE22 (K,M)
FREE - FREE
DOUBLE PRECISION IE11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,

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6FREE55(6,6)
COMMON FR111,FR11,FE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE22=IE11(K,M)
RETURN
END

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END
DOUBLE PRECISION FUNCTION IREE22(K,M)

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C
FREE - FREE

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DOUBLE PRECISION IREE11

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DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)

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COMMON FR111,FR11,FE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE22=IREE11(K,M)
RETURN
END

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DOUBLE PRECISION FUNCTION IRI32 (K,M)

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C
FREE - FREE

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DOUBLE PRECISION IRI11

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DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)

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COMMON FR111,FR11,FE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,

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1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR132=IR11(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION I42 (K,M)

FREE - FREE

DOUBLE PRECISION IR11

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)

I I42=IR11(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE13 (K,M)

FREE - FREE

DOUBLE PRECISION IE11

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,

C

C

3FE55, FREE55, H, XMU, NT(7)

IE13=IE11(K, M)

RETURN

END

DOUBLE PRECISION FUNCTION IR133 (K, M)

FREE - FREE

DOUBLE PRECISION IR111

DOUBLE PRECISION FR111(6,6), FR11(6,6), FE11(6,6), FREE11(6,6), ROB,  
1FR121(6,6), FE21(6,6), FR131(6,6), FE31(6,6), FE12(6,6), FR122(6,6),  
2FR22(6,6), FE22(6,6), FREE22(6,6), FR132(6,6), F42(6,6), FE13(6,6), H,  
3FR133(6,6), FR33(6,6), FE33(6,6), FREE33(6,6), F43(6,6), F53(6,6), SA,  
4FRE53(6,6), FR144(6,6), FR44(6,6), FE44(6,6), FREE44(6,6), FR154(6,6),  
5FE54(6,6), FRE35(6,6), FE45(6,6), FR155(6,6), FR55(6,6), FE55(6,6), PI,  
6FREE55(6,6)

COMMON FR111, FR11, FE11, FREE11, FR121, FE21, FR131, FE31, FE12, FR122, SA,  
1FR22, FE22, FREE22, FR132, F42, FE13, FR133, FR33, FE33, FREE33, F43, F53, PI,  
2FRE53, FR144, FR44, FE44, FREE44, FR154, FE54, FRE35, FE45, FR155, FR55, ROB,  
3FE55, FREE55, H, XMU, NT(7)

IR133=IR111(K, M)

RETURN

END

DOUBLE PRECISION FUNCTION IR33 (K, M)

FREE - FREE

DOUBLE PRECISION IR11

DOUBLE PRECISION FR111(6,6), FR11(6,6), FE11(6,6), FREE11(6,6), ROB,  
1FR121(6,6), FE21(6,6), FR131(6,6), FE31(6,6), FE12(6,6), FR122(6,6),  
2FR22(6,6), FE22(6,6), FREE22(6,6), FR132(6,6), F42(6,6), FE13(6,6), H,  
3FR133(6,6), FR33(6,6), FE33(6,6), FREE33(6,6), F43(6,6), F53(6,6), SA,  
4FRE53(6,6), FR144(6,6), FR44(6,6), FE44(6,6), FREE44(6,6), FR154(6,6),  
5FE54(6,6), FRE35(6,6), FE45(6,6), FR155(6,6), FR55(6,6), FE55(6,6), PI,  
6FREE55(6,6)

COMMON FR111, FR11, FE11, FREE11, FR121, FE21, FR131, FE31, FE12, FR122, SA,  
1FR22, FE22, FREE22, FR132, F42, FE13, FR133, FR33, FE33, FREE33, F43, F53, PI,  
2FRE53, FR144, FR44, FE44, FREE44, FR154, FE54, FRE35, FE45, FR155, FR55, ROB,  
3FE55, FREE55, H, XMU, NT(7)

IR33=IR11(K, M)

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RETURN
END
DOUBLE PRECISION FUNCTION IE33 (K,M)
FREE - FREE
DOUBLE PRECISION IE11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE33=IE11(K,M)
RETURN
END

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C

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DOUBLE PRECISION FUNCTION IREE33(K,M)
FREE - FREE
DOUBLE PRECISION IREE11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE33=IREE11(K,M)
RETURN
END

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C

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C      DOUBLE PRECISION FUNCTION I43 (K,M)
      FREE - FREE
      DOUBLE PRECISION IR11
      DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
      1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
      2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
      3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
      4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
      5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
      6FREE55(6,6)
      COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
      1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
      2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
      3FE55,FREE55,H,XMU,NT(7)
      1 I43=IR11(K,M)
      RETURN
      END
C      DOUBLE PRECISION FUNCTION I53 (K,M)
      FREE - FREE
      DOUBLE PRECISION RHOBAR,CSR,SSR,CDR,SDR,DS,DD
      DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
      1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
      2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
      3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
      4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
      5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
      6FREE55(6,6)
      COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
      1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
      2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
      3FE55,FREE55,H,XMU,NT(7)
      FOR A CYLINDER
      C      IF(SA)6,1,6
      C      1 IF(M-K)5,2,5

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2 IF(M)4,3,4
3 I53 =1.00/ROB
GO TO 18
4 I53 =.500/ROB
GO TO 18
5 I53 =0.00
GO TO 18

C
C
C
FOR A CONE

6 DS=DFLOAT(M+K)
DD=DFLOAT(M-K)
IF(M-K)7,8,7
7 ITAG=2
GO TO 11
8 IF(M)10,9,10
9 I53 =DLOG((ROB+SA)/ROB)/SA
GO TO 18
10 ITAG=1
11 RHOBAR=ROB/SA
XSI=DS*PI*RHOBAR
XSF=DS*PI*(RHOBAR+1.00)
CSR=COS(XSI)
SSR=SIN(XSI)
CALL SICI (SXSI,CXSI,XSI)
CALL SICI (SXSF,CXSF,XSF)
GOTO (14,15),ITAG
14 I53=.500*(+CSR*(CXSF-CXSI)+SSR*(SXSF-SXSI)+DLOG(1.00+SA/ROB))/SA
GO TO 18
15 XDI=DD*PI*RHOBAR
XDF=DD*PI*(RHOBAR+1.00)
CDR=COS(XDI)
SDR=SIN(XDI)
CALL SICI (SXDI,CXDI,XDI)
CALL SICI (SXDF,CXDF,XDF)
I53 =.500*(+CSR*(CXSF-CXSI)+SSR*(SXSF-SXSI)

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1      +CDR*(CXDF-CXDI)+SDR*(SXDF-SXDI))/SA
18 RETURN
END
C      DOUBLE PRECISION FUNCTION IRE53 (K,M)
FREE - FREE
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(M)2,1,2
1 IRE53=0.00
GO TO 4
2 IF(DFLOAT(M+K)/2.00-(M+K)/2)3,1,3
3 IRE53=-2.00*DFLOAT(M*M)/DFLOAT(M*M-K*K)
4 RETURN
END
C      DOUBLE PRECISION FUNCTION IRI44 (K,M)
FREE - FREE
DOUBLE PRECISION IRI1
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,

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3FE55,FREE55,H,XMU,NT(7)
2 IR144=IR11(K,M)
  RETURN
  END
  DOUBLE PRECISION FUNCTION IR44 (K,M)
  FREE - FREE
  DOUBLE PRECISION DS,DD
  DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
  COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
  IF(SA)6,1,6
  1 IF(M-K)5,2,5
  2 IF(M)4,3,4
  3 IR44=ROB*ROB
  GO TO 18
  4 IR44=ROB*ROB/2.D0
  GO TO 18
  5 IR44=0.D0
  GO TO 18
  6 DS=DFLOAT(M+K)*PI
  DD=DFLOAT(M-K)*PI
  IF(M-K)12,10,12
10 IF(M)12,11,12
11 IR44=ROB*ROB+ROB*SA+SA*SA/3.D0
  GO TO 18
12 CONTINUE
  T1=SA*(DCOS(DS)*{(ROB+SA)-ROB})/(DS*DS)
  IF(M-K)14,13,14

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13 T2=.5D0*(ROB*ROB+ROB*SA+SA*SA/3.D0)
GO TO 17
14 T2=SA*(DCOS(DD))*(ROB+SA)-ROB)/(DD*DD)
17 IR44=T1+T2
18 RETURN
END
DOUBLE PRECISION FUNCTION IE44 (K,M)
FREE - FREE
DOUBLE PRECISION DS,DD
DOUBLE PRECISION FR11(6,6),FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FREE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(SA)5,1,5
1 IF(M)3,2,3
2 IE44=0.D0
GO TO 18
3 IF(DFLOAT(M+K)/2.D0-(M+K)/2)4,2,4
4 IE44=-2.D0*ROB*DFLOAT(M*M)/DFLOAT(M*M-K*K)
GO TO 18
5 IF(M)6,2,6
6 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
T1=.5D0*DFLOAT(M)*PI*(DCOS(DD))*(ROB+SA)-ROB)/DS
IF(M-K)10,7,10
7 T2=0.D0
IF(M)9,8,9
8 T3=2.D0*SA
GO TO 17

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9 T3=SA
GO TO 17
10 T2=+.5DD*DFLOAT(M)*PI*(DCOS(DD)*(ROB+SA)-ROB)/DD
T3=0.00
17 IE44=T1+T2+T3
18 RETURN

```

C

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END
DOUBLE PRECISION FUNCTION IREE44(K,M)
FREE - FREE
DOUBLE PRECISION DS,DD,IR44
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IF(M+K)2,1,2
1 IREE44=0.00
GO TO 6
2 DS=DFLOAT(M+K)*PI
DD=DFLOAT(M-K)*PI
T1=-PI*PI*DFLOAT(M*M)*IR44(K,M)
T2=+SA*PI*DFLOAT(M)*(DCOS(DS)*(ROB+SA)-ROB)/DS
IF(M-K)4,3,4
3 T3=0.00
GO TO 5
4 T3=+SA*PI*DFLOAT(M)*(DCOS(DD)*(ROB+SA)-ROB)/DD
5 IREE44=T1+T2+T3
6 RETURN
END
DOUBLE PRECISION FUNCTION IR154 (K,M)

```

C FREE - FREE  
 DOUBLE PRECISION I53  
 DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)  
 COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)

1 IR154=I53(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE54 (K,M)

C FREE - FREE

DOUBLE PRECISION IRE53

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
 6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)

1 IE54=IRE53(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IRE35 (K,M)

C FREE - FREE

DOUBLE PRECISION IRE53

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IRE35=IRE53(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IE45 (K,M)

FREE - FREE

DOUBLE PRECISION IRE53,I53

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,  
6FREE55(6,6)

COMMON FR111,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
3FE55,FREE55,H,XMU,NT(7)

IE45=IRE53(K,M)+SA\*I53(K,M)

RETURN

END

DOUBLE PRECISION FUNCTION IR155 (K,M)

FREE - FREE

DOUBLE PRECISION IR111

DOUBLE PRECISION FR111(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),

2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR155=IR11(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION IR55 (K,M)

C

FREE - FREE

DOUBLE PRECISION IR11

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,  
 4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),  
 5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),PI,  
 6FREE55(6,6)

COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,  
 1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,  
 2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,  
 3FE55,FREE55,H,XMU,NT(7)  
 IR55=IR11(K,M)  
 RETURN  
 END

DOUBLE PRECISION FUNCTION IE55 (K,M)

C

FREE - FREE

DOUBLE PRECISION IE11

DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,  
 1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),  
 2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,  
 3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,

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4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IE55=IE11(K,M)
RETURN
END
DOUBLE PRECISION FUNCTION IREE55(K,M)
FREE - FREE
DOUBLE PRECISION IREE11
DOUBLE PRECISION FR11(6,6),FR11(6,6),FE11(6,6),FREE11(6,6),ROB,
1FR121(6,6),FE21(6,6),FR131(6,6),FE31(6,6),FE12(6,6),FR122(6,6),
2FR22(6,6),FE22(6,6),FREE22(6,6),FR132(6,6),F42(6,6),FE13(6,6),H,
3FR133(6,6),FR33(6,6),FE33(6,6),FREE33(6,6),F43(6,6),F53(6,6),SA,
4FRE53(6,6),FR144(6,6),FR44(6,6),FE44(6,6),FREE44(6,6),FR154(6,6),
5FE54(6,6),FRE35(6,6),FE45(6,6),FR155(6,6),FR55(6,6),FE55(6,6),PI,
6FREE55(6,6)
COMMON FR11,FR11,FE11,FREE11,FR121,FE21,FR131,FE31,FE12,FR122,SA,
1FR22,FE22,FREE22,FR132,F42,FE13,FR133,FR33,FE33,FREE33,F43,F53,PI,
2FRE53,FR144,FR44,FE44,FREE44,FR154,FE54,FRE35,FE45,FR155,FR55,ROB,
3FE55,FREE55,H,XMU,NT(7)
IREE55=IREE11(K,M)
RETURN
END

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C