

LAMINAR FREE CONVECTIVE HEAT TRANSFER IN THE  
SUPERCRITICAL REGION WITH VARIABLE  
FLUID PROPERTIES

By

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## NOMENCLATURE

### English Letter Symbols

A,B,C,D	constants used in Equations (4.15) and (4.26)
A1	defined by Equation (3.16)
A2	defined by Equation (3.17)
A3	defined by Equation (3.18)
$A_2, A_3, A_4, A_5, B_2, B_3, B_5$ $C_2, C_3, C_5, b, K$	constants used in the equation of state, see Equation (4.1)
$A_{11}-A_{55}, B_{11}-B_{55},$ $C_{11}-C_{55}, D_{11}-D_{55}$	constants used in Equations (5.1) through (5.5)
a,b,c,d	constants used in Equations (7.5), (7.10), and (7.18)
$a_1, b_1, c_1$	constants used in Equations (7.8) and (7.17)
$a_0$ through $a_5$	defined by Equation (4.2)
B1	defined by Equation (3.19)
B2	defined by Equation (3.20)
C	defined by Equation (2.4)
$C_p$	specific heat at constant pressure
$C_v$	specific heat at constant volume
$\bar{C}$	reference temperature constant
CA through CD	constants used in Equation (5.7)
CE through CH	constants used in Equation (5.8)
CI through CL	constants used in Equation (5.9)
CM,CN,CP,CR, CS,CT,CU,CW	constants used in Equation (5.10)

CAA through CDD	constants used in Equation (5.11)
CEE through CHH	constants used in Equation (5.12)
c,n	constants used in Equation (7.4)
F( $\eta$ )	similarity variable, defined by Equation (3.7)
Gr	Grashof number, $Gr = [gL^3/\nu^2 (\rho_\infty - \rho_w/\rho_w)]$
g	acceleration due to gravity
h	heat transfer coefficient
K	thermal conductivity
L	overall length of plate
M	molecular weight
Nu	Nusselt number, $Nu = hL/K$
P	pressure
$P_c$	critical pressure
$P_r$	reduced pressure, $P_r = P/P_c$
Pr	Prandtl number
$\dot{q}''$	heat flux
R	gas constant
Ra	Rayleigh number, $Ra = (Pr Gr)$
s	specific entropy
T	temperature
$T_w$	wall or plate temperature
$T_\infty$	free stream temperature
$T_m$	mean temperature
$T_{ref}$ or $T_x$	reference temperature
$T_c$	critical temperature
$T_r$	reduced temperature, $T_r = T/T_c$
$T_M$	pseudocritical or transposed critical temperature

u	velocity component along the plate, or specific internal energy
v	velocity component perpendicular to the plate, or specific volume
X	defined by Equation (4.14)
x	coordinate along the plate, or the parameter in the reference temperature equation, Equation (7.3)
$x_{13}, x_{14}, x_{15}$	constants used in Equation (4.9)
y	coordinate perpendicular to the plate
z	defined by Equation (4.2)
$z_c$	critical compressibility factor, see Equation (4.25)

#### Greek Letter Symbols

$\Delta T$	temperature difference, $\Delta T = T_w - T_\infty$
$\delta$	boundary layer thickness
$\eta$	similarity coordinate as defined by Equation (3.6)
$\lambda$	defined by Equation (4.25)
$\mu$	absolute viscosity
$\nu$	kinematic viscosity, $\nu = \mu/\rho$
$\rho$	density
$\rho_c$	critical density
$\rho_r$	reduced density, $\rho_r = \rho/\rho_c$
$\psi$	stream function, defined by Equation (3.5)
$\theta(\eta)$	dimensionless temperature, defined by Equation (3.8)

#### Subscripts

c	refers to critical property
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c.p.	refers to constant property solution
m	evaluated at the mean temperature
o	refers to zero or low pressure property
P	refers to constant pressure
r	refers to reduced property
ref	evaluated at the reference temperature
$T_x$	evaluated at the temperature $T_x$
v	refers to constant volume
v.p.	refers to variable property solution
w	evaluated at the wall temperature
x	evaluated at a particular point along a surface, a local parameter as opposed to a mean parameter
$\infty$	evaluated at the free stream temperature

#### Symbols for Units

atm	atmosphere
Btu	British thermal units
$^{\circ}\text{C}$	degree Celsius
cal	calorie
cm	centimeter
ft	foot
$^{\circ}\text{F}$	degree Fahrenheit
g	gram
hr	hour
in.	inch
$^{\circ}\text{K}$	degree Kelvin
lbf	pound force
lbm	pound mass

mm	millimeter
psia	pound force per square inch absolute
°R	degree Rankine
sec	second

## CHAPTER I

### INTRODUCTION

Usually, rates of heat transfer by free convection are low compared to forced convection. However, several investigators (1) have observed high rates of free convection heat transfer in the critical and supercritical regions. This is primarily due to the fact that specific heat and compressibility for fluids in the critical and supercritical regions increase significantly when compared to the variation of these properties in the subcritical region.

High rates of convective heat transfer to fluids in the supercritical region has become increasingly important in connection with such applications as the use of helium to cool the coils of superconducting electromagnets and superconducting electronic or power-transmission equipment, the use of hydrogen as a working fluid or fuel for both chemical and nuclear rockets; the use of water in electricity generating plants, and the use of methane as a coolant and fuel for the supersonic transport (1). Free convection is also important in the efficient cooling of turbine blades in high speed and high temperature gas turbines, in cooling of rocket motors by hydrocarbons, in the refrigeration problems, and in cryogenic computer design (2).

The growing applications for high rates of free convection heat transfer to fluids in the supercritical region necessitates the accurate analytical prediction of heat transfer in this region. The analysis for

heat transfer to fluids in the supercritical region is fairly difficult because of the peculiar and relatively large variation of the physical properties of fluids in the supercritical region with temperature, particularly because of the inversion of dependence of constant pressure specific heat upon temperature (see Figures 2 through 6, Appendix B).

In the present study the isothermal, vertical flat plate is chosen as the mathematical model and a steady, two-dimensional laminar boundary layer type of flow is assumed. Many investigators have made analytical investigations of this problem (3) through (7), but the existing prediction schemes are generally inefficient, complicated, and difficult to use. The available prediction methods are not general (i.e., they are dependent on the fluid of interest), and require extensive computer programming. They also require information on the development of analytical procedures for determining the thermodynamic and transport properties including their derivatives for the fluids of interest. A simple, accurate, and practical method for predicting heat transfer to fluids in the supercritical region appeared to be desirable.

The primary objectives of this study were: (1) to develop an accurate, efficient, and general scheme to predict heat transfer to fluids in the supercritical region under variable property conditions in laminar free convection on an isothermal vertical flat plate, and (2) to extend this prediction scheme to develop an accurate, shorthand method for calculating heat transfer to fluids in the supercritical region. This shorthand method involves the use of the results for constant property fluids, and of a rule (reference temperature) for extending these constant property results to variable property situations in the supercritical region.

The problem analysis included the following phases:

1. The application of the equations expressing conservation of mass, momentum, and energy to the problem of free convection under steady state conditions on a vertical flat plate with constant wall and free stream temperatures. These sets of nonlinear partial differential equations with variable coefficients were reduced to a set of two nonlinear ordinary differential equations with variable coefficients by means of the compressible flow stream function and a similarity variable. The variable coefficients were expressed in dimensionless form in terms of the thermodynamic and transport properties (see Chapter III).

2. The development of analytical procedures for determining the thermodynamic and transport properties including their derivatives for Refrigerant-114, water, and carbon dioxide in the supercritical region. The variation with temperature at constant pressure of density, specific heat at constant pressure, absolute viscosity, and thermal conductivity were determined. For most of the fluids encountered in the supercritical region, expressions for physical properties did not exist and were developed from the available experimental data (see Chapter IV).

3. The numerical solution of the reduced ordinary nonlinear differential momentum and energy equations by the method of Lentini-Pereyra (8). This is a variable order finite difference method for nonlinear multipoint boundary value problems. The method is extremely accurate and performs very efficiently. The numerical solution to these differential equations is complicated by the fact that two of the boundary conditions are unknown, namely,  $F''(0)$  and  $\theta'(0)$  (see Chapter V).

4. The development of computer programs to: (a) evaluate transport and thermodynamic properties of Refrigerant-114, water, and carbon



dioxide in the supercritical region; (b) solve the reduced ordinary non-linear differential equations; and (c) calculate heat transfer coefficients for different combinations of wall and free stream temperatures (see Chapter V and Appendix D).

5. The demonstration of the validity and universality of the present variable free convective heat transfer model with the available experimental results and theoretical models for Refrigerant-114, water, and carbon dioxide (see Chapter VI).

6. The development of a correlation for the calculated heat transfer data employing conventional dimensionless groups. The correlation developed was for Refrigerant-114, water, and carbon dioxide applicable in the range of supercritical temperatures and pressures considered in this study (see Chapter VII).

7. The development of an optimization technique that would predict the reference temperature. This was achieved by evaluating the physical properties in the dimensionless groups of the correlation developed for the calculated heat transfer data (see part 6) at an optimum temperature which resulted in the best agreement between the correlated and calculated heat transfer coefficients for certain values of wall and free stream temperatures at different specified constant pressures (see Chapter VII).

8. The development of general plots for reference temperature for different fluids in the supercritical region and wide ranges of temperature differences between the wall and free stream at different constant pressures. These generalized plots were developed for Refrigerant-114, water, and carbon dioxide (see Chapter VII).

9. The demonstration of the validity and universality of the reference temperature method. This was achieved by comparing the constant property heat transfer results obtained using the reference temperature scheme with the available experimental and theoretical property heat transfer results for the three fluids (see Chapter VII).

## CHAPTER II

### LITERATURE SURVEY

In this chapter a brief review of the literature describing the analytical and experimental studies related to this work are presented.

The survey of the work reported in the literature was classified into three different categories. They are:

1. Free Convection
  - (a) Analytical
  - (b) Experimental
2. Physical Properties
3. Reference Temperature.

Each one of the above categories will be presented separately.

#### Free Convection

##### Analytical Work

The first analytical solution to the problem of free convection was obtained by Pohlhausen (9). He was able to show how the partial differential equations of momentum and energy could be transformed into ordinary differential equations. He obtained a solution for the case of air with Prandtl number of 0.73, assuming that the fluid properties are constant with the exception of the role that density variations play in developing buoyancy forces.

Ostrach (10) presented a detailed treatment of the constant property problem for free convection along a vertical flat plate. His paper contained a thorough coverage of the derivations of the equations and presents numerically computed cases for the range of Prandtl numbers from 0.01 to 1000.

The first variable property analytical study was done by Sparrow and Gregg (11). Their work was concerned with free convection along a vertical flat plate. They considered several ideal gases in which a perfect gas type variation was used for density and the constant pressure specific heat; absolute viscosity and thermal conductivity were considered as functions of temperature alone. For absolute viscosity and thermal conductivity a Sutherland type of relation was used. Therefore, their results are for a limited class of fluid behavior.

Fritsch and Grosh's (3) analytical work relates to the problem of free convection heat transfer to supercritical water along a vertical flat plate. In their investigation variations of density and constant pressure specific heat were only considered, while the absolute viscosity and thermal conductivity were assumed to be constant and were evaluated at the mean temperature. The results from their partial variable property analysis for water were as much as 22 percent lower than the experimental results (12).

Fritsch and Grosh concluded that better agreement would have been obtained if account were taken of the variations with temperature of the transport properties as well as the thermodynamic properties. They utilized the assumption of constant absolute viscosity and thermal conductivity to reduce the set of nonlinear partial differential equations to a set of nonlinear differential equations. This procedure has the

effect of eliminating the role that the derivatives of these properties play in their analysis; however, in certain portions of the supercritical region these properties do have strong variations. Because of the restrictions placed on their work, the applicability of their solution in the region above the thermodynamic critical point is not always sufficiently accurate.

Hasegawa and Yoshioka (4) analyzed laminar free convection heat transfer from an isothermal vertical plate for fluids close to the transposed critical point. They calculated the first perturbation solution to the non-perturbated solution for the fluid having physical properties at transposed critical point, expressing all the relevant physical properties as some power functions of enthalpy. Their expressions for physical properties as a function of enthalpy is applicable only for a narrow temperature range. Their results seem to be in qualitative agreement with the experimental results only in case of small temperature differences (12). For large temperature differences, however, the heat transfer coefficient estimated from their analysis was much smaller than those measured. Therefore, the results of their work may be reasonable in a narrow range of temperature difference between wall or free stream and transposed critical point.

The theoretical study of Nishikawa and Ito (6) was made on the laminar free convective heat transfer from an isothermal plate to fluids at supercritical pressures by taking into account the temperature dependence of all relevant physical properties. In their studies the heat transfer characteristics of water and carbon dioxide were clarified by integrating the similarity transformed differential equations numerically. The results of their analysis agrees favorably with the analysis of Fritsch and

Grosh (3), but fails to show good agreement with experimental data of water (12) and carbon dioxide (29) (30).

Parker and Mullin (5) extended the work of Sparrow and Gregg (11) to obtain a method which is more generally useful in all regions of fluid behavior and for all types of fluid in which proper thermodynamic relationships are available. Their technique allows for variation in density, specific heat, absolute viscosity, and thermal conductivity. An important step in the development of their method is the use of thermodynamic relationships to obtain derivatives of properties with respect to temperature. Their method was demonstrated for Refrigerant-114 using a Martin (13) equation of state along with Sutherland and Bromley (14) (15) equations for absolute viscosity and thermal conductivity, corrected for pressure.

An analytical investigation of free convection heat transfer to supercritical water was made by Nowak and Konanur (7). They calculated heat transfer to supercritical water (at 3400 psia in the transposed critical region) by stable laminar free convection from an isothermal, vertical flat plate. In their investigation the actual variations with temperature of all or some of the thermophysical properties of supercritical water were taken into consideration. Fair agreement was found between their analytical values and existing experimental data (12). They concluded that the reasons for lack of complete agreement between their analytical results and the experimental data are due to deficiencies in the existing experimental heat transfer data, and errors in the thermophysical properties of water in the supercritical region.

## Experimental Work

The free convection experimental work related to this study is summarized in Table I of Appendix A. The experimental work presented in this section deals primarily with heat transfer from small test sections, such as wires, filaments, and plates. Table I lists source references for the experimental work, the type of fluid used, the geometry of the apparatus, and the range of conditions at which the experiments were conducted (1).

### Physical Properties

Table II of Appendix A summarized the physical properties of the dominant working fluids in the supercritical region. This table lists source references for equations of state and transport properties. It also includes an indication as to whether the particular authors suggest computation of other useful thermodynamic and transport properties (1).

### Reference Temperature

There are no reported analytical investigations to calculate reference temperature in the supercritical region under variable property conditions in laminar free convection on a vertical flat plate. The utility of the reference temperature lies in the fact that it allows constant property results to be used to compute variable property results.

Parker et al. (5) and Nowak et al. (7) in their work have pointed out that the use of typical reference temperatures in the supercritical region would not produce consistent results. The most widely used reference temperatures are: the surface temperature, the free stream temperature, or a temperature partway between the surface and the free stream.

These reference temperatures produce good results for the problems with moderate property variations. In the supercritical region where there exist large variations in the physical property values compared to the fluids in the subcritical region a more sophisticated method is required.

The only reported analysis in the literature that calculates reference temperature under variable property conditions in laminar free convection on an isothermal vertical flat plate is the work of Sparrow and Gregg (11). Because of the restrictions placed on their work, their suggested reference temperature is not applicable in the entire region above the thermodynamic critical point. They propose that for gases the variable property problem of free convection can be computed using constant property results by evaluating the properties at a reference temperature defined by

$$T_{\text{ref}} = T_w - 0.38 (T_w - T_\infty) \quad (2.1)$$

Since the different gases considered by Sparrow and Gregg did not have large variations in property values compared to the fluids in the critical and supercritical regions, the constant property results for which the property values were evaluated at the reference temperature defined by Equation (2.1) agreed within one percent of the variable property results.

Equation (2.1) is essentially based on a perfect gas state with a power-law and Sutherland-type formula to describe absolute viscosity and thermal conductivity variations. In effect, Sparrow and Gregg found a reference temperature for evaluating thermal conductivity and absolute viscosity such that the constant property solution would yield the correct value for heat transfer coefficient. The constant 0.38 of the



reference temperature Equation (2.1) can be computed using the information in their paper and the following reasoning:

1. The heat transfer for the variable property case is equal to the heat transfer for the constant property case using the reference temperature; that is:

$$\dot{q}_w'' = \dot{q}_{ref}'' \quad (2.2)$$

2. Continuing

$$K_w C_w \theta'(0)_w = K_{ref} C_{ref} \theta'(0)_{ref} \quad (2.3)$$

where

$\theta'(0)_w$  = the dimensionless temperature slope at the wall for the variable property case; and

$\theta'(0)_{ref}$  = the dimensionless temperature slope at the wall for the constant property case.

3. The term C is defined as

$$C = \left[ \frac{g}{4\nu^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right) \right]^{1/4} \quad (2.4)$$

and therefore,

$$\frac{C_{ref}}{C_w} = \left( \frac{\nu_w}{\nu_{ref}} \right)^{1/2} \quad (2.5)$$

4. The substitution of Equation (2.5) into Equation (2.3) gives the necessary equation to be satisfied if Equation (2.2) is to be true; that is:

$$\left( \frac{K_{ref}}{K_w} \right) \left( \frac{\mu_w}{\mu_{ref}} \right)^{1/2} \left( \frac{\rho_{ref}}{\rho_w} \right)^{1/2} = \frac{\theta'(0)_w}{\theta'(0)_{ref}} \quad (2.6)$$

Using Equation (2.6) and the expressions for the physical properties of a given fluid including the values of  $\theta'(0)_w$  and  $\theta'(0)_{ref}$ , one can find an expression to compute the reference temperature constant directly.

Sparrow and Gregg considered the fluid (Gas A)

$$P = \rho R T \quad (2.7a)$$

$$K \sim T^{3/4} \quad (2.7b)$$

$$\mu \sim T^{3/4} \quad (2.7c)$$

$$C_p = \text{constant} \quad (2.7d)$$

$$Pr = \text{constant} \quad (2.7e)$$

The value of  $\bar{C}$ , the constant of the reference temperature Equation (2.1), will now be evaluated for this type of fluid. The various ratios that are required by Equation (2.6) are formed as follows:

$$\frac{\rho_{ref}}{\rho_w} = \frac{T_w}{T_{ref}} \quad (2.8)$$

$$\frac{\mu_w}{\mu_{ref}} = \frac{K_w}{K_{ref}} = \left(\frac{T_w}{T_{ref}}\right)^{3/4} \quad (2.9)$$

Substituting Equations (2.8) and (2.9) into Equation (2.6) yields

$$\frac{T_{ref}}{T_w} = \left(\frac{\theta'(0)_{ref}}{\theta'(0)_w}\right)^8 \quad (2.10)$$

The reference temperature Equation (2.1) may be rearranged as follows:

$$\frac{T_{ref}}{T_w} = 1 - \bar{C} \left(1 - \frac{T_\infty}{T_w}\right) \quad (2.11)$$

The combining of Equations (2.10) and (2.11) and then solving for  $\bar{C}$  gives

$$\bar{C} = \frac{1 - (\theta'(0)_{\text{ref}}/\theta'(0)_w)^8}{1 - (T_\infty/T_w)} \quad (2.12)$$

The results obtained from application of Equation (2.12) to Gas A with  $Pr = 0.7$  are tabulated in Table III (Appendix A). These results were used to evaluate the constant of the reference temperature equation.

Another fluid considered by Sparrow and Gregg was (Gas B)

$$P = \rho R T \quad (2.13a)$$

$$K \sim T^{2/3} \quad (2.13b)$$

$$\mu \sim T^{2/3} \quad (2.13c)$$

$$C_p = \text{constant} \quad (2.13d)$$

$$Pr = \text{constant} \quad (2.13e)$$

If the same procedure that was used for Gas A is applied to Gas B, the following is obtained:

$$\bar{C} = \frac{1 - (\theta'(0)_{\text{ref}}/\theta'(0)_w)^6}{1 - (T_\infty/T_w)} \quad (2.14)$$

The application of Equation (2.14) to Gas B with  $Pr = 0.7$  produces results similar to those of Gas A tabulated in Table III (Appendix A).

Sparrow and Gregg plotted  $T_{\text{ref}}/T_w$  versus  $(T_w - T_\infty)/T_w$  for various values of  $T_w/T_\infty$ , that is, Equation (2.11). The slope of these lines represents the reference temperature constant  $\bar{C}$ . The table of  $\bar{C}$  values for Gas A given in Appendix A confirms the choice of  $\bar{C} = 0.38$  as a good estimate of the constant used for the reference temperature equation.

In view of the types of fluid states assumed by Sparrow and Gregg, they have restricted their results to that portion of the property diagram where their physical property assumptions are valid which is primarily in the perfect gas region. Therefore, the reference temperature equation suggested by Sparrow and Gregg is not valid throughout the supercritical region. This is primarily due to the fact that the parameter  $\bar{C}$  of Equation (2.11) is no longer a constant in this region.

Since heat transfer calculation for the problems relating to free convection along a vertical flat plate in the supercritical region is complicated, tedious, and time-consuming, it would be desirable to use the simple constant property equations to produce the variable property results. The important question is what reference temperature should be used for evaluation of the properties of the constant property problem in order to produce the variable property results. In view of lack of such investigations in the literature, it would be the primary objective of this study to answer this question.

In order to answer the question on the choice of a reference temperature, it is necessary to employ an analytical procedure that predicts heat transfer to fluids in the supercritical region under variable property conditions. The literature search shows that very little analytical work has been done in predicting heat transfer to fluids in the supercritical region considering all the physical property variations. The few existing prediction schemes are inefficient, complicated, difficult to use, and in most cases do not compare well with the experimental data. Therefore, in order to complete the reference temperature analysis, it is also necessary to develop a general, accurate, and reliable method to

predict heat transfer to fluids in the supercritical region under variable property conditions.

## CHAPTER III

### THE FUNDAMENTAL EQUATIONS FOR LAMINAR FREE CONVECTION

It is desired to calculate the nondimensionalized heat transfer coefficient for constant and variable fluid property problems. This chapter outlines the basic equations that were used to solve the constant and variable fluid property problems for laminar free convection on an isothermal vertical flat plate. The necessary mathematical developments that needed to be undertaken in order to generate the reduced set of equations to be handled by the computer program are introduced. Finally, the expressions needed for heat transfer calculations are presented.

#### Physical Model

The physical model and the coordinate system of the present model are shown in Figure 1 (Appendix B). Since it seems easier to visualize occurrences associated with the case where the wall temperature exceeds the free stream temperature, the analysis will be directed toward this situation. However, for the reverse case of the free convection heat transfer problem, a slight change in the coordinate system is necessary, so that the results of the above case are applicable.

In the present model, the lower edge of the plate is chosen as the origin. The distances  $X$  and  $Y$  are measured along and normal to the plate, respectively. The plate extends from  $X$  equal to zero to  $X$  equal

to infinity (i.e., a semi-infinite vertical plate). The direction of the gravity force is taken as pointing downward. This leads to the free convection motion to be upward and away from the leading edge, and the boundary layer thickness at the leading edge to be zero. The velocity components along X and Y directions are u and v, respectively.

### Fundamental Equations

The fundamental equations of conservation of mass, momentum, and energy for laminar free convection in a boundary layer may be written in the following form (see Reference (51)):

#### Mass

$$\frac{\partial}{\partial x} (\rho u) + \frac{\partial}{\partial y} (\rho v) = 0 \quad (3.1)$$

#### Momentum

$$\rho \left( u \frac{\partial u}{\partial x} + v \frac{\partial u}{\partial y} \right) = g(\rho_\infty - \rho) + \frac{\partial}{\partial y} \left( \mu \frac{\partial u}{\partial y} \right) \quad (3.2)$$

#### Energy

$$\rho C_p \left( u \frac{\partial T}{\partial x} + v \frac{\partial T}{\partial y} \right) = \frac{\partial}{\partial y} \left( K \frac{\partial T}{\partial y} \right) \quad (3.3)$$

In accordance with the usual practice in free convection, viscous dissipation and work against the gravity field were neglected. The driving force for the free convection motion is the buoyancy term  $g(\rho_\infty - \rho)$ ; it is formed by combining the pressure gradient  $\partial P/\partial x$  with the body force  $\rho g$ .

The problem will be completely defined by assigning the boundary conditions. The boundary conditions appropriate to the problem are

$$\left. \begin{array}{l} v = 0 \\ u = 0 \\ T = T_w \end{array} \right\} y = 0 \qquad \left. \begin{array}{l} u = 0 \\ T = T_\infty \end{array} \right\} y = \infty \qquad (3.4)$$

In writing the conservation equations and their boundary conditions the following assumptions were made:

- (1) The flow is laminar.
- (2) The flow is steady.
- (3) The flow is two-dimensional.
- (4) The fluid properties are variable.
- (5) The wall temperature is constant.
- (6) The free stream temperature is constant.
- (7) The mechanism of heat transfer is only by conduction and convection.

#### Reduction to Ordinary Differential Equations

Equations (3.1) through (3.3) form a set of nonlinear, nonhomogeneous, simultaneous partial differential equations with variable coefficients, whose exact analytical solutions are unknown. These equations can be reduced to a set of two nonlinear ordinary differential equations by means of the compressible stream function and a similarity variable. The stream function reduces the number of dependent variables, the conservation of mass equation is eliminated. However, it has the disadvantage that the order of the momentum and energy equations are increased. The similarity variable transforms the partial differential equations to ordinary differential equations. The stream function and similarity variable are defined as follows:



### Stream Function

$$u = \frac{\rho_w}{\rho} \left( \frac{\partial \psi}{\partial y} \right) \quad (3.5a)$$

$$v = -\frac{\rho_w}{\rho} \left( \frac{\partial \psi}{\partial x} \right) \quad (3.5b)$$

### Similarity Variable

$$\eta = C_w x^{-1/4} \int_0^y \frac{\rho}{\rho_w} dy \quad (3.6)$$

where

$$C_w = \left[ \frac{g}{4\nu_w^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right)^{1/4} \right]$$

According to the definition of similarity, the new dependent velocity and temperature functions,  $F$  and  $\theta$ , respectively, are assumed to be dependent upon  $\eta$  alone. The new dependent variables  $F$  and  $\theta$  are given by

$$F(\eta) = \left( \frac{1}{4\nu_w C_w} \right) \left( \frac{\psi}{x^{3/4}} \right) \quad (3.7)$$

$$\theta(\eta) = \frac{T - T_\infty}{T_w - T_\infty} \quad (3.8)$$

The function  $F$  is related to the tangential and normal velocities  $u$  and  $v$  and the function  $\theta$  is related to the temperature distribution.

Using the transformation equations, Equations (3.5) through (3.8), we will obtain the set of two nonlinear ordinary differential equations, previously published by Sparrow and Gregg (11).

$$\frac{d}{d\eta} \left( \frac{\rho \mu}{\rho_w \mu_w} F'' \right) + 3FF'' - 2(F')^2 + \frac{(\rho_\infty - \rho)\rho_w}{(\rho_\infty - \rho_w)\rho} = 0 \quad (3.9)$$

$$\frac{d}{d\eta} \left( \frac{\rho K}{\rho_w K_w} \theta' \right) + 3Pr_w \left( \frac{C_p}{C_{p_w}} \right) F\theta' = 0 \quad (3.10)$$

The boundary conditions, Equation (3.4), transform to

$$\left. \begin{array}{l} F = 0 \\ F' = 0 \\ \theta = 1 \end{array} \right\} \eta = 0 \quad \left. \begin{array}{l} F' = 0 \\ \theta = 0 \end{array} \right\} \eta = \infty \quad (3.11)$$

It may be noted that the transformation relations, as well as the resulting ordinary differential equations, can be reduced directly to the well-known equations for the constant property fluid by making the usual approximations. The resulting equations for momentum and energy, respectively, are

$$F''' + 3FF'' - 2(F')^2 + \theta = 0 \quad (3.12)$$

$$\theta'' + 3PrF\theta' = 0 \quad (3.13)$$

The boundary conditions are identical to those given by Equation (3.11). In Equations (3.12) and (3.13), Prandtl number is the only parameter upon which the solution of the equations depends.

The form in which Equations (3.9) and (3.10) exists requires for the physical properties to be expressed as functions of the independent variable  $\eta$  or as functions of the dependent variables  $F$  and  $\theta$ . The mathematical procedures necessary for solving these equations are long and tedious. However, at a particular pressure, the physical properties can be expressed as functions of temperature alone. Since temperature is a function of  $\theta(\eta)$  (see Equation (3.8)), it is an easy matter to obtain the physical properties as functions of  $\theta(\eta)$  once the state equations for the physical properties are known. Manipulation of the

coefficients in Equations (3.9) and (3.10) by the use of certain well-known thermodynamic relationships results in the following equations as previously suggested by Parker and Mullin (5). A sample detailed derivation is given in Appendix C.

#### Momentum Equation

$$F''' = -(A1)\theta'F'' - 3(A2)FF'' + 2(A2)(F')^2 - (A3) \quad (3.14)$$

#### Energy Equation

$$\theta'' = -(B1)(\theta')^2 - 3(B2)F\theta' \quad (3.15)$$

#### Dimensionless Coefficients

$$A1 = \left[ \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p + \frac{1}{\mu} \left( \frac{\partial \mu}{\partial T} \right)_p \right] (\Delta T) \quad (3.16)$$

$$A2 = \frac{\rho_w \mu_w}{\rho \mu} \quad (3.17)$$

$$A3 = (A2) \left[ \frac{\rho_\infty - \rho}{\rho_\infty - \rho_w} \right] \frac{\rho_w}{\rho} \quad (3.18)$$

$$B1 = \left[ \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p + \frac{1}{K} \left( \frac{\partial K}{\partial T} \right)_p \right] (\Delta T) \quad (3.19)$$

$$B2 = (A2) (\text{Pr}) \quad (3.20)$$

The five coefficients A1, A2, A3, B1, and B2 can be determined once the equations of state for the physical properties  $C_p$ ,  $\mu$ ,  $K$ , and  $\rho$  are known as functions of temperature and pressure. The analytical solutions to the reduced differential Equations (3.14) and (3.15) which satisfy the boundary conditions given by Equation (3.11) are not known and were obtained by numerical methods on a computer (see Chapter V). These differential equations were solved simultaneously due to the appearance of common terms in the two differential equations (i.e., the differential

equations are coupled). Physically, this is due to the fact that the fluid motions in free convection are caused by density variations which in turn are due to the temperature gradients in the fluids.

### Heat Transfer at the Wall

The local heat flux from the wall (plate) to the fluid may be calculated using Fourier's law:

$$\dot{q}_{x,w}'' = -K_w \left( \frac{\partial T}{\partial y} \right)_{y=0} \quad (3.21)$$

Introducing the dimensionless variables from Equations (3.6) through (3.8) in Equation (3.21), the local heat transfer rate in terms of the solutions of the differential equations becomes

$$\dot{q}_{x,w}'' = -K_w (T_w - T_\infty) C_w x^{-1/4} \theta'(0) \quad (3.22)$$

where  $\theta'(0)$ , the slope of the dimensionless temperature profile at the wall, is found from the solutions of Equations (3.14) and (3.15) for the variable property case, or Equations (3.12) and (3.13) for the constant property case, along with the boundary conditions given by Equation (3.11).

The local heat transfer coefficient is defined by the expression

$$h_x = \frac{\dot{q}_{x,w}''}{(T_w - T_\infty)} \quad (3.23)$$

A dimensionless representation of the local heat transfer coefficient in terms of the solution of the differential equations is achieved by defining a local Nusselt number and a local Grashof number. The

defined Grashof number is applicable to both constant and variable property case.

$$Nu_{x,w} = \frac{h_x x}{K_w} \quad (3.24)$$

$$Gr_{x,w} = \left[ \frac{gx^3}{\nu_w^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right) \right] \quad (3.25)$$

Using these definitions of  $h_x$ ,  $Nu_{x,w}$ , and  $Gr_{x,w}$ , the local heat flux given by Equation (3.22) becomes

$$Nu_{x,w} = - \frac{Gr_{x,w}^{1/4}}{\sqrt{2}} \theta'(0) \quad (3.26)$$

Dimensionless heat transfer coefficients on a vertical flat plate in laminar free convection are usually presented in the form of Equation (3.26).

## CHAPTER IV

### PHYSICAL PROPERTIES

In this chapter the need for the expressions for physical properties as functions of pressure and temperature is established. The availability of these expressions is reviewed. The analytical procedures for determining these expressions are explained. Finally, the expressions developed for physical properties based on the available experimental data are presented.

In order to solve the free convection heat transfer problem (either constant or variable property), the accurate knowledge of both the thermodynamic and transport properties is necessary. That is, before the solutions to the reduced form of the momentum and energy equations, Equations (3.14) and (3.15) for the variable property case or Equations (3.12) and (3.13) for the constant property case, can be obtained it is necessary to determine the five dimensionless coefficients  $A_1$ ,  $A_2$ ,  $A_3$ ,  $B_1$ , and  $B_2$  expressed by Equations (3.16) through (3.20). These coefficients require the knowledge of the following thermodynamic and transport properties:

<u>Thermodynamic Properties</u>	<u>Symbols</u>
1. Density	$\rho$
2. Coefficient of volume expansion	$\frac{1}{\rho} (\partial\rho/\partial T)_p$
3. Specific heat at constant pressure	$C_p$
<u>Transport Properties</u>	<u>Symbols</u>
1. Absolute viscosity	$\mu$

- |                         |  |
|-------------------------|--|
| 2. The term             | $\frac{1}{\mu} (\partial\mu/\partial T)_p$ |
| 3. Thermal conductivity | K  |
| 4. The term             | $\frac{1}{K} (\partial K/\partial T)_p$    |

The values of the above properties can be determined for any temperature at a particular pressure if the following information is known:

1. An equation of state for the fluid of interest.
2. An expression for the specific heat at constant pressure as a function of temperature.
3. An expression for the absolute viscosity as a function of temperature.
4. An expression for the thermal conductivity as a function of temperature.

The most widely used working fluids in the supercritical region are halocarbon compounds (Refrigerants), water, carbon dioxide, helium, hydrogen, nitrogen, oxygen, propane, and ammonia (see Table II, Appendix A). Refrigerant-114 was the only working fluid for which all of the above information was available in the region where both pressures and temperatures were above critical. On the other hand, for most of the widely used fluids in the supercritical region, there is enough experimental thermodynamic and transport property data available in the literature (see Table II, Appendix A). Therefore, based on the available experimental data, it was possible to develop the analytical procedures for determining the required expressions for the physical properties.

Determination of the four required expressions for physical properties to start the heat transfer calculations is a very difficult task in itself. This complication is due to the peculiar variations of the physical properties of fluids in the supercritical region. For

illustrative purposes Figures 2 through 5 (Appendix B) show variations in the thermodynamic and transport properties of supercritical water with respect to temperature at a constant pressure of 3400 psia.

Once the types of equations that would represent the physical properties were recognized, it was possible to develop the analytical procedures necessary for determination of the physical property expressions based on the experimental data. A computer program called MARQ developed by Jackson and Chandler (52) was adopted for this purpose. This program performs a nonlinear least squares fit of a user-supplied function to a given set of data, using Marquardt-S method, or the Gauss-Newton method, or a modified Gauss-Newton method. A complete documented listing of the MARQ program is given in Appendix D.

To demonstrate the universality of the present model, it was necessary to test the method with different fluids in the supercritical region. The fluids chosen for this study were Refrigerant-114, water, and carbon dioxide. These fluids were chosen due to the availability of reliable physical property expressions and experimental data (see Table II, Appendix A), and also existence of theoretical and experimental heat transfer data (see Table I, Appendix A).

To start the solutions of the reduced differential equations, physical property expressions were developed for water and carbon dioxide. These expressions along with the thermodynamic and transport properties of Refrigerant-114, compiled and developed by Mullin (50), are the equation of state, coefficient of volume expansion, specific heat at constant pressure, absolute viscosity, the term  $[1/\mu(\partial\mu/\partial T)_p]$ , thermal conductivity, and the term  $[1/K(\partial K/\partial T)_p]$ .



## Equation of State

Martin (13) developed a twelve parameter equation of state for Refrigerant-114. This equation, which is valid in the supercritical region, was used to determine the density as a function of temperature at constant pressure. The same type of equation was used in the MARQ program to fit the pressure-volume-temperature experimental data of supercritical water (40) (41) (53) and supercritical carbon dioxide (54) (55) (56). The fit to the given set of data was excellent. The average absolute errors between the fitted results and the experimental data for both fluids were less than 2 percent.

The equation of state developed by Martin (13) is as follows:

$$\begin{aligned}
 P = & \frac{RT}{(V-b)} + \frac{A_2 + B_2T + C_2 \text{EXP}(-K T_r)}{(V-b)^2} \\
 & + \frac{A_3 + B_3T + C_3 \text{EXP}(-K T_r)}{(V-b)^3} + \frac{A_4}{(V-b)^4} \\
 & + \frac{A_5 + B_5T + C_5 \text{EXP}(-K T_r)}{(V-b)^5}
 \end{aligned} \tag{4.1}$$

where

$$T_r = \frac{T}{T_c}$$

The units and constants to be used in Equation (4.1) for all three fluids (Refrigerant-114, water, and carbon dioxide) are tabulated in Table IV (Appendix A).

To obtain a value of  $\rho = \rho(T)$  at constant pressure it was necessary to use numerical techniques due to the fact that the equation of state is explicit only in pressure. The procedure is as follows:

1. Define the following terms:

$$z = (V - b) \quad (4.2a)$$

$$a_0 = P \quad (4.2b)$$

$$a_1 = RT \quad (4.2c)$$

$$a_2 = A_2 + B_2T + C_2 \text{ EXP}(-K T_r) \quad (4.2d)$$

$$a_3 = A_3 + B_3T + C_3 \text{ EXP}(-K T_r) \quad (4.2e)$$

$$a_4 = A_4 \quad (4.2f)$$

$$a_5 = A_5 + B_5T + C_5 \text{ EXP}(-K T_r) \quad (4.2g)$$

2. The equation of state, Equation (4.1), was rewritten as follows:

$$f(z) = a_0 z^5 - a_1 z^4 - a_2 z^3 - a_3 z^2 - a_4 z - a_5 = 0 \quad (4.3)$$

3. The first derivative of the 5th degree polynomial, Equation (4.3), is:

$$f'(z) = 5a_0 z^4 - 4a_1 z^3 - 3a_2 z^2 - 2a_3 z - a_4 \quad (4.4)$$

4. The problem was then reduced to finding the proper root of the polynomial for a specified pressure and temperature. The Newton-Raphson (57) iteration method was used to obtain the roots. This equation has the form:

$$z_{n+1} = z_n - \frac{f(z)}{f'(z)} \quad (4.5)$$

5. After the proper root for Equation (4.3) was obtained, the density was computed from the following expression:

$$\rho = \frac{1}{(z + b)} \quad (4.6)$$

The entire procedure was programmed for the IBM 370 computer. The procedure is part of the physical property subroutines developed for Refrigerant-114, water, and carbon dioxide (see Appendix D).

Having established the equations of state for the fluids of interest in the supercritical region, the next step in evaluation of the dimensionless coefficients of the reduced differential equations, expressed by Equations (3.16) through (3.20), was determination of the coefficient of volume expansion.

$$\text{Coefficient of Volume Expansion, } 1/\rho \left(\frac{\partial \rho}{\partial T}\right)_P$$

An expression for the coefficient of volume expansion was obtained as follows:

1. Equation of state, Equation (4.1), is implicit in volume and temperature. Therefore, it was helpful to start with the techniques of implicit differentiation given by the following expression:

$$\left(\frac{\partial V}{\partial T}\right)_P \left(\frac{\partial T}{\partial P}\right)_V \left(\frac{\partial P}{\partial V}\right)_T = -1 \quad (4.7)$$

2. An expression for the coefficient of volume expansion in terms of the derivatives of the equation of state was obtained by rearrangement of Equation (4.7). This equation had the advantage that the dependent variable of the derivatives was pressure. The resulting equation was

$$\frac{1}{\rho} \left(\frac{\partial \rho}{\partial T}\right)_P = \rho \frac{(\partial P/\partial T)_V}{(\partial P/\partial V)_T} \quad (4.8)$$

where

$$V = \frac{1}{\rho}$$

and

$$\left(\frac{\partial P}{\partial T}\right)_V = \frac{R}{(V-b)} + \frac{B_2 - (K/T_c) C_2 \text{EXP}(-K T_r)}{(V-b)^2} + \frac{B_3 - (K/T_c) C_3 \text{EXP}(-K T_r)}{(V-b)^3} + \frac{B_5 - (K/T_c) C_5 \text{EXP}(-K T_r)}{(V-b)^5}$$

and

$$\left(\frac{\partial P}{\partial V}\right)_T = -\frac{RT}{(V-b)^2} - \frac{2[A_2 + B_2 T + C_2 \text{EXP}(-K T_r)]}{(V-b)^3} - \frac{3[A_3 + B_3 T + C_3 \text{EXP}(-K T_r)]}{(V-b)^4} - \frac{4A_4}{(V-b)^5} - \frac{5[A_5 + B_5 T + C_5 \text{EXP}(-K T_r)]}{(V-b)^6}$$

These equations were programmed for the IBM 370 computer. The above equations are part of the physical property subroutines given in Appendix D. The last thermodynamic expression to be developed was the specific heat at constant pressure.

#### Specific Heat at Constant Pressure

Along with an equation of state for Refrigerant-114, Martin (13) also developed the following expression for the specific heat at constant volume and zero pressure for Refrigerant-114.

$$C_{V_0} = X_{13} + X_{14}T + X_{15}T^2 \quad (4.9)$$

The units and constants to be used in Equation (4.9) are tabulated in Table V (Appendix A).

Reynolds and Perkins (58) indicate that the same type of equation as given above would also express the specific heat at constant volume and zero pressure for other supercritical fluids, namely water and carbon dioxide. With this information and the use of the equation of state and the thermodynamic potential functions, it was possible to derive an equation for the specific heat at constant pressure. An examination of the thermodynamic relation

$$C_p - C_v = T \left( \frac{\partial P}{\partial T} \right)_V \left( \frac{\partial V}{\partial T} \right)_P \quad (4.10)$$

shows that expressions for the right-hand side may be obtained from the equation of state. Since an expression for the specific heat at constant volume and zero pressure was known, the problem was then reduced to obtaining an expression for the effects of pressure on the value of the specific heat at constant volume. This was accomplished by the use of thermodynamic potential functions and Maxwell equations. The end result is given by Equation (4.11).

$$C_v = C_{v_0} - T \left( \frac{K}{T_c} \right)^2 \text{EXP}(-K T_r) \left[ \frac{C_2}{(V-b)} + \frac{C_3}{2(V-b)^2} + \frac{C_5}{4(V-b)^4} \right] \quad (4.11)$$

For a detailed derivation of Equation (4.11) see Appendix C. Equation (4.11) was used to calculate the effect of pressure change in the specific heat at constant volume for a given temperature.

The specific heat at constant pressure was then obtained by substituting Equations (4.9) and (4.11) into Equation (4.10), and then solving Equation (4.10) for  $C_p$ , the specific heat at constant pressure. The resulting equation for specific heat calculations was

$$\begin{aligned}
C_P = & X_{13} + X_{14}T + X_{15}T^2 \\
& - T\left(\frac{K}{T_c}\right) \text{EXP}(-K T_r) \left[\frac{C_2}{(V-b)} + \frac{C_3}{2(V-b)^2} + \frac{C_5}{4(V-b)^4}\right] \\
& + T\left(\frac{\partial P}{\partial T}\right)_V \left(\frac{\partial V}{\partial T}\right)_P
\end{aligned} \tag{4.12}$$

The above equation was employed in the MARQ program to fit the specific heat at constant pressure experimental data of supercritical water (37) (38) (40) (41) (59) and supercritical carbon dioxide (46) through (49). At this stage it was found that simultaneous fit of experimental specific volume and specific heat data would result in a better fit, when compared to the fit with experimental specific heat data alone. This could be primarily due to the errors involved in the experimental thermodynamic property measurements of different investigators. Considering the peculiar variation of specific heat at constant pressure with respect to temperature in the supercritical region (see Figure 6, Appendix B), the fit to the given set of data was considered to be very good. The average absolute errors between the fitted results and the experimental data for both fluids were less than 4 percent. Equation (4.12) was also used by Mullin (50) to calculate specific heat at constant pressure for Refrigerant-114.

The units and constant to be used in Equation (4.12) for all three fluids (Refrigerant-114, water, and carbon dioxide) are tabulated in Table V (Appendix A). Equation (4.12) was programmed for the IBM 370 computer and is part of the physical property subroutines given in Appendix D.

Having established the required expressions for the thermodynamic properties, the next step was development of the necessary equations for

the transport properties. The first transport property considered was the absolute viscosity.

### Absolute Viscosity

For Refrigerant-114, Parker and Mullin (5) developed an expression for viscosity at low pressure by the use of a Sutherland-type (60) viscosity equation and low pressure viscosity experimental data of Downing (61):

$$\mu_0 = 1.0167 \times 10^{-3} \left[ \frac{T^{3/2}}{T + 586.55} \right] \quad (4.13)$$

The viscosity data from the above equation was then corrected for pressure based on the equation developed by Jossi et al. (14):

$$(\mu - \mu_0)X = [23.12 \text{ EXP}(1.079 \rho_r) - 25.0] \times 10^{-5} \quad (4.14)$$

where

$$X = \frac{T_c^{1/6}}{M^{1/2} P_c^{2/3}}$$

and

$$\rho_r = \frac{\rho}{\rho_c}$$

The units and constants to be used in Equations (4.13) and (4.14) are given in Table VI (Appendix A).

For water and carbon dioxide, due to the availability of good density and viscosity experimental data, it was possible to develop the following simple correlation:

$$\mu = A + BT + C\rho^D \quad (4.15)$$

The above equation was developed based on the correlations reported by Fritsch et al. (3) (39) and Nowak et al. (59) for water. Equation (4.15) was used in the MARQ program to fit the absolute viscosity experimental data of supercritical water (39) (40) (41) and supercritical carbon dioxide (42) (43). The fit to the given set of data was excellent. The average absolute errors between the fitted results and the experimental data for both fluids were less than 1.5 percent.

The units and constants to be used in the viscosity equations for all three fluids (Refrigerant-114, water, and carbon dioxide) are tabulated in Table VI (Appendix A). These equations were programmed for the IBM 370 computer and are part of the physical property subroutines given in Appendix D.

Having established the absolute viscosity expressions for the fluids of interest in the supercritical region, the next step in evaluation of the dimensionless coefficients of the reduced differential equations, expressed by Equations (3.16) through (3.20), was determination of the term  $1/\mu (\partial\mu/\partial T)_P$ .

The Term  $1/\mu (\partial\mu/\partial T)_P$

An expression for the term  $1/\mu (\partial\mu/\partial T)_P$  was obtained employing the following equations:

$$\mu = \mu(\rho, T) \quad (4.16)$$

$$\frac{1}{\mu} \left( \frac{\partial\mu}{\partial T} \right)_P = \frac{1}{\mu} \left[ \left( \frac{\partial\mu}{\partial\rho} \right)_T \left( \frac{\partial\rho}{\partial T} \right)_P + \left( \frac{\partial\mu}{\partial T} \right)_\rho \right] \quad (4.17)$$

Performing the operations indicated by Equation (4.17) on Equation (4.15) and the equation of state, Equation (4.1), resulted in the following expressions for water and carbon dioxide:



$$\left(\frac{\partial \mu}{\partial \rho}\right)_T = (C)(D)(\rho)^{(D-1)} \quad (4.18)$$

$$\left(\frac{\partial \rho}{\partial T}\right)_P = \rho^2 \frac{(\partial P / \partial T)_V}{(\partial P / \partial V)_T} \quad (4.19)$$

$$\left(\frac{\partial \mu}{\partial T}\right)_\rho = B \quad (4.20)$$

Substitution of Equations (4.15), (4.18), (4.19), and (4.20) into Equation (4.17) resulted in the desired transport property expressions for water and carbon dioxide.

The same procedure was used to evaluate the term  $1/\mu (\partial \mu / \partial T)_P$  for Refrigerant-114. In this case Equations (4.13) and (4.14), along with the equation of state were used. The following expressions for Refrigerant-114 were obtained:

$$\left(\frac{\partial \mu}{\partial \rho}\right)_T = \frac{1.079}{\rho_c} \left[ (\mu - \mu_0) + \frac{2.5 \times 10^{-4}}{\lambda} \right] \quad (4.21)$$

$$\left(\frac{\partial \rho}{\partial T}\right)_P = \rho^2 \frac{(\partial P / \partial T)_V}{(\partial P / \partial V)_T} \quad (4.22)$$

$$\left(\frac{\partial \mu}{\partial T}\right)_\rho = \left(\frac{3}{2}\right) \frac{\mu_0}{T} - \left(\frac{\mu_0}{T + 586.55}\right) \quad (4.23)$$

Substitution of Equations (4.13), (4.14), (4.21), (4.22), and (4.23) in Equation (4.17) resulted in the desired transport property expression for Refrigerant-114.

These equations were programmed for the IBM 370 computer and are part of the physical property subroutines given in Appendix D. The last transport property expression to be developed was the thermal conductivity.

## Thermal Conductivity

For Refrigerant-114, Parker and Mullin (5) developed an expression for thermal conductivity at low pressure based on the equation recommended by Downing (61):

$$\frac{MK_0}{\mu_0} = 0.0132 C_{V_0} + 0.034 - \frac{0.007}{T_r} \quad (4.24)$$

The thermal conductivity data from the above equation was then corrected for pressure based on the equation developed by Stiel and Thodos (37):

$$(K - K_0) \lambda Z_C^5 = 13.10 \times 10^{-8} [\text{EXP}(0.67 \rho_r) - 1.069] \quad (4.25)$$

where

$$\lambda = \frac{M^{1/2} T_C^{1/6}}{p_C^{2/3}}$$

and

$$Z_C = \frac{p_C}{RT_C \rho_C}$$

The units and constants to be used in Equations (4.24) and (4.25) are given in Table VII (Appendix A).

For water and carbon dioxide, due to the availability of good density and thermal conductivity experimental data, it was possible to develop the following simple correlation:

$$K = A + BT + C\rho^D \quad (4.26)$$

The above equation was developed based on the correlations reported by Fritsch et al. (3) (39) and Nowak et al. (59) for water. Equation (4.26)

was used in the MARQ program to fit the thermal conductivity experimental data of supercritical water (39) (40) (41) and supercritical carbon dioxide (44) (45). The fit to the given set of data was very good. The average absolute errors between the fitted results and the experimental data for both fluids was less than 3 percent.

The units and constants to be used in the thermal conductivity equations for all three fluids (Refrigerant-114, water, and carbon dioxide) are tabulated in Table VII (Appendix A). These equations were programmed for the IBM 370 computer and are part of the physical property subroutines given in Appendix D.

The last transport property to be determined for the evaluation of the dimensionless coefficients A1, A2, A3, B1, and B2, expressed by Equations (3.16) through (3.20), was the term  $1/K (\partial K/\partial T)_p$ .

#### The Term $1/K (\partial K/\partial T)_p$

An expression for the term  $1/K (\partial K/\partial T)_p$  was obtained in a manner similar to that used to obtain  $1/\mu (\partial \mu/\partial T)_p$ . The following equations were employed:

$$K = K(\rho, T) \quad (4.27)$$

$$\frac{1}{K} \left( \frac{\partial K}{\partial T} \right)_p = \frac{1}{K} \left[ \left( \frac{\partial K}{\partial \rho} \right)_T \left( \frac{\partial \rho}{\partial T} \right)_p + \left( \frac{\partial K}{\partial T} \right)_\rho \right] \quad (4.28)$$

Performing the operations indicated by Equation (4.28) on Equation (4.26) and the equation of state, Equation (4.1), resulted in the following expressions for water and carbon dioxide:

$$\left( \frac{\partial K}{\partial \rho} \right)_T = (C)(D)(\rho)^{(D-1)} \quad (4.29)$$

$$\left(\frac{\partial \rho}{\partial T}\right)_P = \rho^2 \frac{(\partial P / \partial T)_V}{(\partial P / \partial V)_T} \quad (4.30)$$

$$\left(\frac{\partial K}{\partial T}\right)_\rho = B \quad (4.31)$$

Substitution of the Equations (4.26) and (4.29) through (4.31) in Equation (4.28) resulted in the desired transport property expressions for water and carbon dioxide.

The same procedure was used to evaluate the term  $1/K (\partial K / \partial T)_\rho$  for Refrigerant-114. In this case Equations (4.24) and (4.25), along with the equation of state were used. The following expressions for Refrigerant-114 were obtained:

$$\left(\frac{\partial K}{\partial \rho}\right)_T = \frac{0.67}{\rho_C} \left[ (K - K_0) + \frac{14.003 \times 10^{-8}}{\lambda Z_C^5} \right] \quad (4.32)$$

$$\left(\frac{\partial \rho}{\partial T}\right)_P = \rho^2 \frac{(\partial P / \partial T)_V}{(\partial P / \partial V)_T} \quad (4.33)$$

$$\begin{aligned} \left(\frac{\partial K}{\partial T}\right)_\rho &= 0.0132 C_{V_0} \left(\frac{\partial \mu_0}{\partial T}\right)_\rho + 0.0132 \mu_0 \left(\frac{\partial C_{V_0}}{\partial T}\right)_\rho \\ &+ \frac{0.034}{M} \left(\frac{\partial \mu_0}{\partial T}\right)_\rho - \frac{0.007}{M T_r} \left(\frac{\partial \mu_0}{\partial T}\right)_\rho + \frac{0.007}{M T_r} \left(\frac{\mu_0}{T}\right) \end{aligned} \quad (4.34)$$

where

$$\left(\frac{\partial \mu_0}{\partial T}\right)_\rho = \frac{3}{2} \left(\frac{\mu_0}{T}\right) - \left(\frac{\mu_0}{T + 586.55}\right)$$

and

$$\left(\frac{\partial C_{V_0}}{\partial T}\right)_\rho = 3.49 \times 10^{-4} - 3.34 \times 10^{-7} T$$

Substitution of Equations (4.24), (4.25), and (4.32) through (4.34) into Equation (4.28) resulted in the desired transport property expression for

Refrigerant-114. These equations were programmed for the IBM 370 computer and are part of the physical property subroutines given in Appendix D.

The thermodynamic and transport property expressions developed for water and carbon dioxide in this chapter showed to be in very good agreement with the available physical property experimental data. These equations appear to represent the data within their experimental accuracy. In obtaining these expressions, as the situation permitted, a special effort was made to keep these equations as simple as possible.

The major difficulty encountered in obtaining these expressions was the discontinuity in the experimental data. In most cases the supercritical experimental physical properties reported in the literature did not cover a wide range of temperature and pressure discretely. This was especially true for the absolute viscosity, thermal conductivity, and specific heat data. Due to the peculiar variations of the physical properties of the fluids in the supercritical region (see Figures 2 through 6, Appendix B), this lack of continuity in the data made the fitting process more complicated.

In general, as one would expect, for a wide range of temperatures and pressures, the accuracy of the physical property equations would increase as one moves away from the critical point. The greatest deviation, of course, occurs in the neighborhood of the critical point, where the physical properties change is extremely rapid with temperature.

The physical property equations developed and presented in this chapter for water, carbon dioxide, and Refrigerant-114 are summarized in Table VIII (Appendix A), along with the information relative to the development of these expressions. This table lists source references

for the experimental data on the thermodynamic and transport properties. It also includes an indication as to the number of data points used in the development of the equations, the percent deviation between the fitted results and the experimental data, and the range of application of these equations. Some of this information was not available for Refrigerant-114.

It should be noted that the units used in the equations for the physical properties of Refrigerant-114 are inconsistent in the sense that different units were used to express the same property. This is due to the different choices made by different investigators. Since the coefficients of the reduced differential equations, Equations (3.14) and (3.15), are dimensionless, the equations were developed in the same system of units as the work reported in the literature and the necessary unit changes were made in the computer program. A complete documented listing of the physical property subroutines for Refrigerant-114, water, and carbon dioxide are given in Appendix D. These programs compute the transport and thermodynamic properties for the indicated fluids in the supercritical region. The programs also compute the five dimensionless coefficients  $A_1$ ,  $A_2$ ,  $A_3$ ,  $B_1$ , and  $B_2$ , expressed by Equations (3.16) through (3.20), needed for the solutions of the reduced differential Equations (3.14) and (3.15), for the variable property case.

## CHAPTER V

### NUMERICAL METHOD FOR THE SOLUTION OF THE REDUCED EQUATIONS

It will be the purpose of this chapter to explain how the set of reduced ordinary nonlinear differential equations introduced in Chapter III were solved.

The analytical investigations reported in the literature for the problem of free convection heat transfer to fluids in the supercritical region employ the classical fourth order Runge-Kutta numerical integration technique (2) (3) (5) (6) (7). This method was used by the investigators to solve the set of reduced ordinary nonlinear differential Equations (3.14) and (3.15) for the variable property case, along with their boundary conditions, Equation (3.11). Konanur (2) in his work points out that the accuracy of the Runge-Kutta method of integration depends upon the step size,  $\Delta\eta$ . The smaller values of  $\Delta\eta$  introduce round-off errors and large values of  $\Delta\eta$  give rise to truncation errors. Hence, for any problem the step size should be estimated by trial integrations. In general, the primary disadvantages of the Runge-Kutta integration method are (62):

1. The method requires significantly more computer time than other methods of comparable accuracy.
2. Local error estimates are somewhat difficult to obtain.
3. The method is not very accurate.

Solution by the Runge-Kutta method is possible if all the necessary boundary values are known. In the present problem, at the first boundary ( $\eta$  equal to zero) out of five required initial boundary values ( $F$ ,  $F'$ ,  $F''$ ,  $\theta$ , and  $\theta'$ ) only three of them ( $F$ ,  $F'$ , and  $\theta$ ) are known. Hence, the other two unknown boundary values ( $F''$  and  $\theta'$ ) have to be estimated. Now the solution of the present problem is one of finding the accurate unknown values. Once the initial values are known, one can proceed with the integration process. For the problem under study the necessary five starting boundary conditions are (see Equation (3.11)):

$$F(0) = 0$$

$$F'(0) = 0$$

$$F''(0) = \text{Unknown}$$

$$\theta(0) = 1$$

$$\theta'(0) = \text{Unknown.}$$

The unknown conditions  $F''$  and  $\theta'$  at the boundary are very important because they fix the shear stress and heat transfer at the plate, respectively. Therefore, they must be determined very accurately. In the works reported in the literature, different iterative schemes were employed to overcome this difficulty. Fritsch et al. (3) and Konanur (2) in their work carry out integration of the reduced equations; they first estimate values of  $F''$  and  $\theta'$  at the first boundary ( $\eta$  equal to zero) and then proceed to a large value of  $\eta$  to check the results with the desired boundary conditions. Then, using an iterative technique, the estimated values were then adjusted until suitable agreement between the calculated and the stipulated values were noted. In order for their method to converge at a reasonable amount of computer time and with reasonable accuracy, good estimates of the unknown boundary conditions are



required. In the absence of good estimates the method is very inefficient. Konanur (2) also points out that the success of his model depends upon the case under study and upon how close the guessed values are to required values. Parker and Mullin (5) used the iterative method of Newton-Raphson to predict the unknown boundary conditions. Their method requires linearization of the reduced ordinary nonlinear differential equations and fairly good estimates for the starting boundary conditions. The above mentioned numerical procedures used by different investigators to solve the type of problem under consideration in this study produce fairly reasonable results, but their drawbacks are:

1. The numerical procedures are inefficient.
2. The numerical methods are not very accurate.
3. The numerical programs are unadaptable. These programs were exclusively written for one type of problem, and usage of these programs for similar type problems requires major modifications.

In pursuit of the primary objectives of this study it was necessary to adopt a numerical procedure which is easy to use, efficient, very accurate, and general. The numerical technique developed by Lentini and Pereyra (8) was used for this purpose. This is an adaptive finite difference method for first order nonlinear systems of ordinary differential equations subject to multipoint nonlinear boundary conditions. The method is based on a discretization studied earlier by H. B. Keller (63). In this method variable order is provided through deferred corrections, while a built-in natural asymptotic estimator is used to automatically refine the mesh in order to achieve a required tolerance. This method is easy to use, extremely accurate, and it performs very efficiently in comparison with the results published in the literature for multipoint,

nonlinear boundary value problems (8). Moreover, the solutions of the reduced differential equations based on this method do not depend upon an initial estimate of the unknown boundary conditions. A thorough and complete discussion of this method is given in References (8) and (64).

#### Evaluation of the Coefficients for the Reduced Differential Equations

At this point it is necessary to explain the method used to evaluate the coefficients of the reduced equations. This is a necessary step before the procedure used to solve the set of reduced ordinary nonlinear equations can be outlined. Since it was desired to express the coefficients of the reduced differential equations as a function of  $\theta(\eta)$  (see Chapter III), it was necessary to make a decision as to what method should be used to evaluate the dimensionless coefficients given by Equations (3.16) through (3.20). It appeared that the following two possible methods were available:

1. The most desirable method would be the evaluation of the dimensionless coefficients of the reduced differential equations by an exact method. This method required writing of the numerical method in such a way that for a given  $\eta$  it permitted the integration method to evaluate  $\theta(\eta)$  at an  $\eta$  which in turn allowed the evaluation of the temperature. Then, with this information, evaluation of the dimensionless coefficients given by Equations (3.16) through (3.20) was made possible. However, it was found that the computer time required for this procedure was totally impractical. Due to the iterative nature of the problem under study, to perform a few iterations without achieving convergence, the procedure required several hours of computer time.

2. The second possibility would be to first compute the values of the dimensionless coefficients, Equations (3.16) through (3.20), for a given temperature range. Then, using a polynomial, curve fit the data by the method of least squares. It is then possible to obtain an expression for the dimensionless coefficients as a function of  $\theta(\eta)$ . This method will introduce some error due to the imperfection of the least squares fit; however, it has the advantage of converging rapidly.

In view of the above experience the second method was used. This method was implemented by first expressing the physical properties shown below as follows:

$$\left[ \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p + \frac{1}{\mu} \left( \frac{\partial \mu}{\partial T} \right)_p \right] = A_{11} + B_{11}T + C_{11}T^2 + D_{11}T^3 \quad (5.1)$$

$$\left[ \frac{1}{\rho \mu} \right] = A_{22} + B_{22}T + C_{22}T^2 + D_{22}T^3 \quad (5.2)$$

$$\left[ \frac{1}{\rho^2 \mu} \right] = A_{33} + B_{33}T + C_{33}T^2 + D_{33}T^3 \quad (5.3)$$

$$\left[ \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p + \frac{1}{K} \left( \frac{\partial K}{\partial T} \right)_p \right] = A_{44} + B_{44}T + C_{44}T^2 + D_{44}T^3 \quad (5.4)$$

$$\left[ \frac{C_p}{\rho K} \right] = A_{55} + B_{55}T + C_{55}T^2 + D_{55}T^3 \quad (5.5)$$

The coefficients for the above polynomials were determined by the method of least squares. A complete documented listing of the least squares program is given in Appendix D. For Refrigerant-114, water, and carbon dioxide used in this study, a third degree polynomial was the highest degree of polynomial required to result in the best possible fit. As mentioned earlier, the method introduced small errors in the calculations due to the imperfection of the least squares fit. At times in the neighborhood of the pseudocritical point this error for the term  $[C_p/\rho K]$ ,

given by Equation (5.5), became significant. This could be mainly contributed to the peculiar variation of the specific heat for fluids in the supercritical region with respect to temperature in that region (see Figure 6, Appendix B). However, it was possible to minimize the error by using smaller temperature intervals for this case. For illustrative purposes, Figures 7, 8, and 9 (Appendix B) show plots of Equations (5.1) through (5.5). The substitution of the expression

$$T = T_{\infty} + \Delta T \theta(\eta) \quad (5.6)$$

into Equations (5.1) through (5.5), and the substitution of Equations (5.1) through (5.5) into Equations (3.16) through (3.20) will give the desired equations for the dimensionless coefficients of the reduced differential Equations (3.14) and (3.15) as functions of  $\theta(\eta)$ . As a result the following expressions were obtained:

$$A1 = CA + CB \theta + CC \theta^2 + CD \theta^3 \quad (5.7)$$

$$3(A2) = CE + CF \theta + CG \theta^2 + CH \theta^3 \quad (5.8)$$

$$2(A2) = CI + CJ \theta + CK \theta^2 + CL \theta^3 \quad (5.9)$$

$$A3 = (CM + CN \theta + CP \theta^2 + CR \theta^3) \\ - (CS + CT \theta + CU \theta^2 + CW \theta^3) \quad (5.10)$$

$$B1 = CAA + CBB \theta + CCC \theta^2 + CDD \theta^3 \quad (5.11)$$

$$3(B2) = CEE + CFF \theta + CGG \theta^2 + CHH \theta^3 \quad (5.12)$$

The coefficients for Equations (5.7) through (5.12) are given in Appendix C. Equations (5.7) through (5.10) given above represent the dimensionless coefficients of the momentum equation, Equation (3.14), and the Equations (5.11) and (5.12) express the dimensionless coefficients of the energy equation, Equation (3.15). The advantage of the method used is that no derivatives of the polynomials are necessary in order to evaluate

the dimensionless property coefficients of the reduced differential equations.

The substitution of Equations (5.7) through (5.12) into Equations (3.14) and (3.15) permit the reduced differential equations to be expressed as follows:

Momentum Equation

$$\begin{aligned}
 F''' = & -(CA + CB \theta + CC \theta^2 + CD \theta^3) \theta' F'' \\
 & - (CE + CF \theta + CG \theta^2 + CH \theta^3) FF'' \\
 & + (CI + CJ \theta + CK \theta^2 + CL \theta^3) (F')^2 \\
 & - (CM + CN \theta + CP \theta^2 + CR \theta^3) \\
 & + (CS + CT \theta + CU \theta^2 + CW \theta^3)
 \end{aligned} \tag{5.13}$$

Energy Equation

$$\begin{aligned}
 \theta'' = & -(CAA + CBB \theta + CCC \theta^2 + CDD \theta^3) (\theta')^2 \\
 & - (CEE + CFF \theta + CGG \theta^2 + CHH \theta^3) F \theta'
 \end{aligned} \tag{5.14}$$

The next step in the development of the reduced differential equations for the computer program was to form a set of first order differential equations to represent the third order momentum differential equation, Equation (5.13), and the reduced second order energy equation, Equation (5.14). The following procedures were used:

1. Define:

$$G_1 = F \tag{5.15a}$$

$$G_2 = F' \tag{5.15b}$$

$$G_3 = F'' \tag{5.15c}$$

$$G_4 = \theta \tag{5.15d}$$

$$G_5 = \theta' \tag{5.15e}$$

2. The set of differential equations, Equations (5.13) and (5.14), become:

$$Y_1 = \frac{dG_1}{d\eta} = G_2 \quad (5.16a)$$

$$Y_2 = \frac{dG_2}{d\eta} = G_3 \quad (5.16b)$$

$$\begin{aligned} Y_3 = \frac{dG_3}{d\eta} = & -(CA + CB G_4 + CC G_4^2 + CD G_4^3) G_5 G_3 \\ & - (CE + CF G_4 + CG G_4^2 + CH G_4^3) G_1 G_3 \\ & + (CI + CJ G_4 + CK G_4^2 + CL G_4^3) (G_2)^2 \\ & - (CM + CN G_4 + CP G_4^2 + CR G_4^3) \\ & + (CS + CT G_4 + CU G_4^2 + CW G_4^3) \end{aligned} \quad (5.16c)$$

$$Y_4 = \frac{dG_4}{d\eta} = G_5 \quad (5.16d)$$

$$\begin{aligned} Y_5 = \frac{dG_5}{d\eta} = & -(CAA + CBB G_4 + CCC G_4^2 + CDD G_4^3) (G_5)^2 \\ & - (CEE + CFF G_4 + CGG G_4^2 + CHH G_4^3) G_1 G_5 \end{aligned} \quad (5.16e)$$

The boundary conditions, Equation (3.11), transform to

$$\left. \begin{array}{l} G_1 = 0 \\ G_2 = 0 \\ G_3 = \text{Unknown} \\ G_4 = 1 \\ G_5 = \text{Unknown} \end{array} \right\} \eta = 0 \quad \left. \begin{array}{l} G_1 = \text{Unknown} \\ G_2 = 0 \\ G_3 = \text{Unknown} \\ G_4 = 0 \\ G_5 = \text{Unknown} \end{array} \right\} \eta = \infty \quad (5.17)$$

Equation (5.16) along with the boundary condition, Equation (5.17), are the set of first order nonlinear differential equations of the

reduced momentum and energy differential equations to be solved on the computer.

### Procedures for Solving the Differential Equations

The procedures and various programs used in solving the set of reduced ordinary nonlinear differential equations are outlined here. The nondimensionalized heat transfer coefficients were then obtained using the solutions to these equations. The procedures used consisted of five different computer programs. They are defined as follows:

1. The Physical Property Program - Subroutine PROP
2. The Least Squares Program - Subroutine POLY
3. The Coefficient Program - Subroutine COEFF
4. The Lentini-Pereyra Program - Subroutine SYSSOL
5. The Heat Transfer Program - Subroutine HEAT.

A complete listing of these programs with some sample calculations are given in Appendix D. These programs are well documented; therefore, a brief discussion of each program will be given here.

#### The Physical Property Program (Subroutine PROP)

This program is a FORTRAN representation of the thermodynamic and transport property equations developed in Chapter IV. The purpose of this program was to compute physical properties and their derivatives for the fluids in the supercritical region at a constant pressure, and a given temperature range. The program was written very generally and requires only four pieces of information. They are:

1. Pressure (PSIA).
2. Initial Temperature ( $^{\circ}$ R).

3. Final Temperature ( $^{\circ}\text{R}$ ).
4. An initial estimate for the specific volume ( $\text{ft}^3/\text{lbm}$ ).

The initial value of the specific volume needs only to be an estimated value, since its only purpose was to start the iteration process of Newton-Raphson on the equations of state (see Chapter IV).

For Refrigerant-114, water, and carbon dioxide used in this study three independent Physical Property Programs were developed. A complete documented listing of these programs are given in Appendix D.

The Physical Property Program also computes the following information:

$$\left[ \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p + \frac{1}{\mu} \left( \frac{\partial \mu}{\partial T} \right)_p \right] \text{ versus } T$$

$$\left[ \frac{1}{\rho \mu} \right] \text{ versus } T$$

$$\left[ \frac{1}{\rho^2 \mu} \right] \text{ versus } T$$

$$\left[ \frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_p + \frac{1}{K} \left( \frac{\partial K}{\partial T} \right)_p \right] \text{ versus } T$$

$$\left[ \frac{C_p}{\rho K} \right] \text{ versus } T$$

The above data obtained from the Physical Property Program provides the necessary information to compute the coefficients for Equations (5.1) through (5.5) with the help of the Least Squares Program.

#### The Least Squares Program (Subroutine POLY)

The purpose of the Least Squares Program was to determine the regression coefficients for Equations (5.1) through (5.5), based on the information obtained from the Physical Property Program. The program calculates the regression coefficients and analysis of variance table



for polynomials of successively increasing degrees. The program also automatically terminates when there is no reduction in the residual sum of squares between two successive degrees of the polynomials; and calculates a residual table for the given and fitted data. This termination occurs before the analysis for the highest degree of polynomial specified is completed. The following information is required to start the Least Squares Program:

1. Independent and dependent variables--for the problem under study these were the physical property versus temperature information obtained from the Physical Property Program.
2. Total number of observations.
3. The highest degree of polynomial desired--for the problem under study third degree or lower.

The following coefficients for Equations (5.1) through (5.5) were computed by the Least Squares Program:

$$\left[ \begin{array}{l} A_{11}, B_{11}, C_{11}, D_{11} \\ A_{22}, B_{22}, C_{22}, D_{22} \\ A_{33}, B_{33}, C_{33}, D_{33} \\ A_{44}, B_{44}, C_{44}, D_{44} \\ A_{55}, B_{55}, C_{55}, D_{55} \end{array} \right]$$

These coefficients provide the necessary information to compute the coefficients of Equations (5.7) through (5.12) given in Appendix C. These coefficients are computed in the Coefficient Program described next.

#### The Coefficient Program (Subroutine COEFF)

The purpose of this program was to determine the coefficients for

Equations (5.7) through (5.12). These equations represent the property coefficients of the reduced differential equations as functions of  $\theta(\eta)$ . These coefficients were evaluated from equations presented in Appendix C for the terms A1, 3(A2), 2(A2), A3, B1, and 3(B2). In order to evaluate the desired coefficients the following information is required:

1. The polynomial coefficients for Equations (5.1) through (5.5), which is obtained from the Least Squares Program.
2. Wall and free stream temperatures, which is obtained from the statement of the problem.
3. The wall viscosity, the wall and free stream density, which is obtained from the Physical Property Program.

The following coefficients for Equations (5.7) through (5.12) were computed by the Coefficient Program.

CA, CB, CC, CD
CE, CF, CG, CH
CI, CJ, CK, CL
CM, CN, CP, CR
CS, CT, CV, CW
CAA, CBB, CCC, CDD
CEE, CFF, CGG, CHH

These coefficients provide the necessary information for the solution of the reduced differential equations by the Lentini-Pereyra Program.

#### The Lentini-Pereyra Program (Subroutine SYSSOL)

The purpose of this program was to solve Equation (5.16) along with the boundary conditions, Equation (5.17). These are the set of first

order nonlinear differential equations of the reduced momentum and energy equations. In order to solve the differential equations by the Lentini-Pereyra Program (Subroutine SYSSOL), the following information is required:

1. The coefficients of Equation (5.16) or Equations (5.7) through (5.12), which is obtained from the Coefficient Program.
2. The set of first order differential equations, Equation (5.16), which is represented by Subroutine FF2.
3. The Jacobian of these first order differential equations, which is represented by Subroutine JACOB2 (for equations, see Appendix C).

In addition to the above mentioned subroutines, the driver program for the Lentini-Pereyra Program (Subroutine SYSSOL), requires the following information:

1. The boundary conditions, Equation (5.17), a value of zero was assumed for the unknown boundary conditions.
2. Number of points in the initial mesh, counting the end points the value should be greater than 3.
3. User's desired accuracy, this specifies the final absolute error of the estimated values.

This program is very general, accurate, easy to use, efficient, and only requires information which is related to the problem statement. Moreover, a good estimate of the unknown boundary conditions is not essential for rapid convergence of the iterative scheme built in the program. A complete documented listing of the program is given in Appendix D. The solutions of the differential equations obtained by the Lentini-Pereyra Program were designated as

$$F(\eta) \quad \theta(\eta)$$

$$F'(\eta) \quad \theta'(\eta)$$

$$F''(\eta) \quad \theta''(\eta)$$

$$F'''(\eta)$$

The Lentini-Pereyra Program was also used to obtain the solutions to the constant property momentum and energy Equations (3.12) and (3.13) and their boundary conditions Equation (3.11). Since the coefficients of the differential equations are all constants, the numerical solution for this case was much simpler in comparison to the variable property case. For this case the Lentini-Pereyra Program requires the same type of information as the ones outlined for the variable property case, with the exception of the Coefficient Program. The only parameter upon which the solution of the constant property case depends is Prandtl number, which is obtained through the Physical Property Program. The constant property results were used in the development of the reference temperature method.

For the constant property case, the set of first order differential equations and their Jacobians used in the Subroutine FF1 and Subroutine JACOBI are presented in Appendix C.

The solutions of the differential equations obtained from the Lentini-Pereyra Program were used to calculate heat transfer coefficients through the Heat Transfer Program.

#### The Heat Transfer Program (Subroutine HEAT)

This program was used to evaluate the local heat flux, the local heat transfer coefficient, the local Grashof number, the local dimensionless heat transfer coefficient (local Nusselt number), and the wall and

free stream Prandtl numbers. The necessary information to perform these calculations were obtained from the Physical Property Program, the Lentini-Pereyra Program, and the problem statement. The following equations introduced earlier in Chapter III were the necessary equations to perform these calculations:

1. Local Heat Flux

$$\dot{q}_{x,w}'' = -K_w (T_w - T_\infty) C_w x^{-1/4} \theta'(0)$$

where

$$C_w = \left[ \frac{g \rho_w^2}{4 \mu_w^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right) \right]^{1/4}$$

In the above equation,  $\theta'(0)$  is the slope of the dimensionless temperature profile at the wall. This information was provided by the Lentini-Pereyra Program from the solutions of the reduced differential equations.

2. Local Heat Transfer Coefficient

$$h_x = \frac{\dot{q}_{x,w}''}{(T_w - T_\infty)}$$

3. Local Wall Grashof Number

$$Gr_{x,w} = \left[ \frac{g x^3 \rho_w^2}{\mu_w^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right) \right]$$

4. Local Wall Nusselt Number

$$Nu_{x,w} = - \frac{Gr_{x,w}^{1/4}}{\sqrt{2}} \theta'(0)$$

## 5. Wall and Free Stream Prandtl Numbers

$$\text{Pr}_w = \frac{\mu_w c_{p_w}}{K_w}$$

$$\text{Pr}_\infty = \frac{\mu_\infty c_{p_\infty}}{K_\infty}$$

All computer programs described in this chapter are located in Appendix D.

Several different problems relating to free convection along a vertical flat plate in the supercritical region for Refrigerant-114, water, and carbon dioxide were solved using the equations and procedures presented in this work. The results will be used in the next chapters to develop the reference temperature method proposed in this study.

## CHAPTER VI

### HEAT TRANSFER RESULTS

In this chapter the validity and universality of the present variable property heat transfer model is established. The present model is compared with several theoretical and experimental investigations for Refrigerant-114, water, and carbon dioxide, and the results are discussed. Finally, the numerical solutions of the present model for the constant property case are compared with the well-known constant property analysis of Ostrach (10).

In order to complete the analysis of the reference temperature method, it was necessary to establish the validity and universality of the present variable property heat transfer model for fluids in the supercritical region. This was accomplished by comparing the solutions of the present model with the former theoretical and experimental studies. Comparisons were made with the theoretical investigation of Parker and Mullin (5) for Refrigerant-114, experimental investigation of Fritsch and Grosh (12) for water, theoretical investigation of Nowak and Konanur (7) for water, experimental investigations of Nishikawa et al. (30) and Kato et al. (29) for carbon dioxide, and theoretical investigation of Nishikawa and Ito (6) for carbon dioxide.

Parker and Mullin (5) solved four different problems relating to free convection along a vertical flat plate in the supercritical region. These problems were solved for Refrigerant-114 at a pressure of 540 psia and a

temperature difference ( $\Delta T = T_w - T_\infty$ ) of 5°F for four values of the wall temperature, 305, 310, 315, and 320°F. Values of the local heat flux,  $\dot{q}_x''$ , versus wall temperature,  $T_w$ , for the analysis of Parker and Mullin are presented in Figure 10 (Appendix B). This figure also shows the results of comparison between the present analytical model and the analytical investigation of Parker and Mullin (5). As shown in Figure 10, the heat transfer results of this model are in excellent agreement with the theoretical results of Parker and Mullin. In this analysis the same physical property expressions as the ones used by Parker and Mullin for Refrigerant-114 were used. However, the numerical methods for the solutions of the differential equations were completely different (see Chapter V). In view of the above experience, it could be concluded that the excellent agreement between the two models is a good indication as to the validity of the present numerical technique. In comparison to the model of Parker and Mullin, the present model proved to be more general, less complicated, and easier to use.

In conjunction with Figure 10, it is important to note that the maximum value of local heat flux occurs at a wall temperature of 310°F, and a free stream temperature of 305°F. These temperatures are very close to the temperature at which specific heat assumes its maximum, that is, transposed critical temperature ( $T_M = 306^\circ\text{F}$ ). This increase in the rate of heat flux is mainly due to the direct and significant influence of the specific heat maximum on heat transfer calculations for the cases where the wall and free stream temperatures are close to  $T_M$ , the transposed critical temperature. This temperature also had a direct and significant effect in the development of the reference temperature method. The reference temperature method will be discussed in the next chapter.



Fritsch and Grosh (12) measured heat transfer data to supercritical water. Their experimental measurements were made in laminar free convection from a vertical flat plate, for water close to its critical point. The point-by-point comparison of the predicted heat transfer fluxes with the experimental data of Fritsch and Grosh is shown in Figure 11. The agreement with the experimental data is considered to be very good.

The lack of complete agreement between the present model and the experimental heat transfer data of Fritsch and Grosh (12) for supercritical water may, in part, be attributable to: (a) the fact that the Fritsch and Grosh experimental heat transfer data were obtained from a surface more closely resembling constant heat flux conditions than one at a uniform temperature. A constant heat flux surface gives higher heat fluxes and heat transfer coefficients than those from a corresponding isothermal heat transfer surface; (b) the fact that the chosen height of the vertical heating wall (1/2 in.) was so small that Rayleigh numbers in their experiments were sufficiently small, but their results could not be said to be an experiment for a vertical wall; and (c) the errors in the physical properties of water in the supercritical region. Figure 12 (Appendix B) compares the predicted values of the Prandtl numbers with the experimental data of Fritsch and Grosh (12). Prandtl number was the only reported experimental physical property data. This figure shows that the predicted Prandtl numbers are higher than the experimental data, but since Prandtl number by definition is the ratio of the product of absolute viscosity and specific heat at constant pressure over thermal conductivity ( $Pr = \mu C_p / K$ ), it is not clear which one of these physical properties is in disagreement with the experimental data.

In reference to the experimental heat transfer data of Fritsch and Grosh (12) there was no indication as to the effect of radiation on their reported heat transfer results for the cases where the difference between the wall and the free stream temperatures were large. In view of lack of such knowledge, calculations were made to include the effect of radiation heat transfer. Based on the calculations made for the experimental data of Fritsch and Grosh, the rate of heat transfer due to radiation proved to be negligible.

The local heat fluxes predicted by the present model are compared to the predictions given by the theoretical model of Nowak and Konanur (7), and the experimental data of Fritsch and Grosh (12) for supercritical water in Figure 13 (Appendix B). It seems from Figure 13 that the present model predicts the experimental data consistently better than the analytical model of Nowak and Konanur. The superiority of the present model becomes more significant for the cases where the difference between the wall and the free stream temperatures become large.

Nishikawa et al. (30) and Kato et al. (29) measured heat transfer data to supercritical carbon dioxide. Their experimental measurements were made in laminar free convection from a vertical flat plate, for carbon dioxide close to its critical point. The comparison between the predicted local heat fluxes and the experimental data is shown in Figure 14 (Appendix B). The predictions of the present model show excellent agreement with the experimental data of Nishikawa et al. (30) for carbon dioxide. However, the present prediction scheme did not predict the experimental data of Kato et al. (29) for carbon dioxide very well. This lack of agreement is due to the fact that in the cases of Kato et al. Rayleigh numbers in their experiments were too large ( $10^9 \sim 10^{12}$ ) to

regard their results as experiments in the laminar regime. However, the height of the vertical walls in their apparatus was fairly large (20 mm) and their results could be considered to be an experiment for a vertical wall.

The results of comparison between the present analytical model and the theoretical model of Nishikawa and Ito (6) for supercritical carbon dioxide is also shown in Figure 14. This figure shows that the present model predicts the experimental data of Nishikawa et al. (30) far better than the analytical model of Nishikawa and Ito (6). The superiority of the present model becomes more pronounced for the cases where the difference between the wall and the free stream temperatures become large.

In general, the present prediction model appears to be in very good agreement with the available experimental data of fluids in the supercritical region. Aside from the specific points that were mentioned in the discussion of the results, in general the lack of complete agreement between the existing model and the experimental data might be attributable to (a) errors in the thermodynamic and transport properties of fluids in the supercritical region, and to (b) the deficiencies in the existing experimental heat transfer data. Among many other items which must be examined, the technique to measure the temperature of the heating wall were not suitable in some experiments, and pressure vessels in some apparatus were too small to realize the condition of stagnant fluid in infinitely large spaces. Accordingly, in some cases, it would be premature to discuss the validity of the analysis by comparison with the available experimental data. The present model in all cases proved to be better than the existing theoretical models in predicting the experimental heat transfer data for fluids in the supercritical region.

The superiority of the present model was far more pronounced for cases where the difference between the wall and the free stream temperatures become large. In contrast to the theoretical models of Nowak and Konanur (7) for water, and Nishikawa and Ito (6) for carbon dioxide, the present model does not specifically require a knowledge of the experimental values of the physical properties of fluids in the supercritical region and their derivatives for the problem under study. The present model also proved to be more general (i.e., it is independent of the fluid of interest), less complicated, easier to use, and more accurate in comparison to the specific models of Nowak and Kananur (7) for water, Nishikawa and Ito (6) for carbon dioxide, and Parker and Mullin (5) for Refrigerant-114.

Figure 15 compares the theoretical values of local heat fluxes predicted by the present model with the experimental values of supercritical water and carbon dioxide. The correlation between the theoretical and experimental values is reasonable and all of the supercritical free convection heat transfer data are predicted by the theory within an error of  $\pm 20$  percent. This shows that the present model is reasonably good at predicting heat transfer to fluids in the supercritical region under variable property conditions in laminar free convection on a vertical flat plate.

The utility of the reference temperature lies in the fact that it allows constant property results to be used for variable properties. Therefore, before introducing the development of the reference temperature method (see Chapter VII), it is proper to prove the validity of the present model in predicting the constant property results. This was achieved by comparing the results of the present prediction technique with the well-known constant property analysis of Ostrach (10). The most

important results to be considered in the constant property analysis for the problem under study are the velocity and temperature distributions.

By means of the various transformations made in the analysis as presented in Chapter III, it can easily be verified that for the constant property case

$$F'(\eta) = \frac{ux/v_w}{2\sqrt{Gr_{x,w}}} \quad (6.1)$$

and

$$\theta(\eta) = \frac{T - T_\infty}{T_w - T_\infty} \quad (6.2)$$

where

$$\eta = \left(\frac{Gr_{x,w}}{4}\right)^{1/4} \frac{y}{x} \quad (6.3)$$

The dimensionless velocity and temperature distributions as given by Equations (6.1) and (6.2) are presented in Figures 16 and 17 (Appendix B), respectively, as functions of  $\eta$ . In these figures the velocity and temperature profiles predicted by the present model are compared with the analysis of Ostrach (10) for  $Pr = 10$ . The computations made with the present model agree with the results of Ostrach up to the fourth significant figure. The excellent agreement between the two analyses proves the validity of the present model in predicting the constant property results.

In reference to the above calculations, it should be noted that in computation of the velocity and temperature profiles, for both constant and variable fluid properties, the value of  $\eta$  equal to 10 was used to satisfy the second boundary conditions at infinity (see Equation (3.11)).

In all the cases considered in this study the established value of infinity never exceeded  $\eta = 8$ ; however, to be on the safe side infinity was defined as  $\eta = 10$ .

## CHAPTER VII

### REFERENCE TEMPERATURE METHOD

This chapter will outline the development of a shorthand method by which heat transfer can be evaluated. In the latter part of this chapter the reference temperature results for three fluids will be presented and discussed.

From Chapters III, IV, and V of this study it can be readily seen that solution to the problems relating to variable property conditions is a long, complicated, and tedious process. It is hoped to eliminate most of this tedious work by using an accurate shorthand method for heat transfer calculations, a technique referred to as the reference temperature method.

For engineering applications it has been found convenient to employ the constant property analytic solutions, or the experimental data obtained with small temperature differences, and then to apply some kind of correction to account for property variations. Most of the variable property results indicate that fairly simple corrections will generally suffice over a moderate range of properties. For the problems in which the absolute value of the properties varies markedly through the boundary layer (see Figures 2 through 6, Appendix B), no simple correction scheme is available and thus a complete numerical integration of the applicable differential equations for each such application will be required. The

problem of free convection along a vertical flat plate in the supercritical region under variable property conditions falls into this category.

The most extensively used scheme for correction of the constant property results is the reference temperature method. In this method a characteristic temperature is chosen at which the properties appearing in the nondimensional groups (Nu, Gr, Pr, etc.) may be evaluated. The constant property results at that temperature may be used to evaluate variable property behavior. Typically this temperature may be the surface temperature, the free stream temperature, or a temperature partway between the surface and the free stream; there is no general rule.

On the basis of this conception, reference temperature can be expressed as follows:

$$T_{\text{ref}} = T_w - \bar{C} (T_w - T_\infty) \quad (7.1)$$

In the analytical investigations reported in the literature for reference temperature, the parameter  $\bar{C}$  of Equation (7.1) has been assumed to have a constant value. This assumption is valid only when the thermodynamic and transport properties of the problem do not vary markedly through the boundary layer, that is, when variations in the physical properties at constant pressure can be assumed linear for small change in temperature. In this case determination of the "reference temperature constant," parameter  $\bar{C}$ , is a fairly simple matter (see Chapter II).

For the present study the above assumption is not valid. Since the absolute value of the thermodynamic and transport properties varies peculiarly through the boundary layer (see Figures 2 through 6, Appendix B). The parameter  $\bar{C}$  is no longer a constant and may be a complicated function of a dimensionless temperature group expressed as follows:



$$x = f(T_M - T_\infty / T_W - T_\infty) \quad (7.2)$$

$T_M$  is the pseudocritical or transposed critical temperature, normally defined as the temperature where the specific heat at constant pressure attains a maximum for a given supercritical pressure (see Figure 3, Appendix B). Then, the reference temperature expression given by Equation (7.1) becomes

$$T_x = T_W - x(T_W - T_\infty) \quad (7.3)$$

An expression similar to Equation (7.2) was suggested by Eckert (65) in an attempt to correlate the near critical, forced convection heat transfer data obtained in heated-tube experiments.

Prior to the present study there appeared to be no analysis available to predict the reference temperature for the problem of free convection along a vertical flat plate in the supercritical region under variable property conditions. It was the primary objective of this study to make this prediction possible. To achieve this goal five different steps had to be accomplished. In order to aid better understanding of the steps taken in the development of the reference temperature method, Refrigerant-114 at a single supercritical pressure of 540 psia was chosen for illustrative purposes. In the latter part of this chapter the results of the following steps were applied to several fluids at different supercritical pressures in order to prove the validity and universality of the present reference temperature method. The steps taken in the development of the reference temperature method were:

1. Calculation of heat transfer coefficients. With the use of the equations, procedures, and computer programs introduced and developed in Chapters III through V, the local dimensionless heat transfer coefficients

(local Nusselt number) were calculated for several cases. For illustrative purposes Refrigerant-114 was chosen as the working fluid and heat transfer data were obtained for 123 different cases, covering a wide range of temperature differences between the wall and the free stream. The range of temperatures covered in these calculations were from 300°F to 350°F at a constant pressure of 540 psia.

2. Development of a correlation for the calculated dimensionless heat transfer data, using data obtained from step 1. The conventional constant property correlation of laminar free convective heat transfer such as

$$Nu = c(Gr Pr)^n = c(Ra)^n \quad (7.4)$$

where Rayleigh number  $Ra$  is taken as the independent variable and Nusselt number  $Nu$  as the dependent variable fails to correlate the heat transfer data in the supercritical region. This is mainly due to the strong temperature dependency of the physical properties.

The following basic form was employed to correlate the heat transfer data for the variable property conditions:

$$Nu_{x,w} = a(Gr_{x,w})^b (Pr_\infty)^c \left( \frac{T_\infty}{T_w - T_\infty} \right)^d \quad (7.5)$$

where the constants  $a$ ,  $b$ ,  $c$ , and  $d$  of Equation (7.5) were obtained from the MARQ program. For the 123 physical cases considered for Refrigerant-114 at 540 psia pressure, the following values were obtained for the constants  $a$ ,  $b$ ,  $c$ , and  $d$  of Equation (7.5):

$$a = 0.40248195$$

$$b = 0.25$$

$$c = 0.31833951$$

$$d = 0.00169742.$$

The constants given above fit the 123 data points with an average absolute error of less than 1 percent.

3. Development of a correlation for constant property results. The utility of the reference temperature method lies in the fact that it allows use of constant property results to compute variable property values. Therefore, it was essential to include the constant property analysis in determination of the reference temperature. The slope of the temperature profile at the wall  $\theta'(0)$  used in these analyses was obtained from the solution of the following constant property momentum and energy equations:

$$F''' + 3FF'' - 2(F')^2 + \theta = 0 \quad (7.6)$$

$$\theta'' + 3PrF\theta' = 0 \quad (7.7)$$

In Equations (7.6) and (7.7), Prandtl number is the single parameter of the equations. For the pressures and temperatures in this study, Prandtl number covered a range from 0 to 40. For this range of Prandtl numbers the following correlation was obtained for  $\theta'(0)$  as a function of Prandtl number:

$$[-\theta'(0)]_{c.p.} = a_1 + b_1(Pr)^{c_1} \quad (7.8)$$

where

$$a_1 = 0.10;$$

$$b_1 = 0.48615754; \text{ and}$$

$$c_1 = 0.33141328.$$

These constants were obtained using the MARQ program. Equation (7.8)

with the constant given above predicts the slope of the temperature profile at the wall for the constant property cases within an average absolute error of 1.3 percent.

Based on the equations presented in Chapter III, the slope of the temperature profile at wall,  $\theta'(0)$ , can be expressed in terms of the dimensionless groups Nusselt number and Grashof number as follows:

$$[-\theta'(0)]_{c.p.} = \frac{Nu_{x,w}}{(Gr_{x,w}/4)^{1/4}} \quad (7.9)$$

4. Development of an optimization procedure to predict reference temperature. For this purpose Equation (7.5) was represented as

$$(Nu_{x,w})_{T_x} = a(Gr_{x,w})_{T_x}^b (Pr)_{T_x}^c \left(\frac{T_\infty}{T_w - T_\infty}\right)^d \quad (7.10)$$

where the properties in the Nusselt number, Grashof number, and Prandtl number were evaluated at a characteristic temperature  $T_x$ , which is a complicated function of wall, free stream, and transposed critical temperatures. The value of  $T_x$  was then optimized between the free stream and wall temperatures until the best agreement between the correlated values,  $(Nu_{x,w})_{T_x}$  and  $[Nu_{x,w}/(Gr_{x,w}/4)^{1/4}]_{T_x}$ , obtained from Equation (7.10) at temperature  $T_x$ , and the calculated values of  $(Nu_{x,w})$  and  $[-\theta'(0)]_{c.p.} = Nu_{x,w}/(Gr_{x,w}/4)^{1/4}$  were reached. This optimum temperature is the reference temperature for that case. In this comparison the calculated Nusselt numbers  $(Nu_{x,w})$  were obtained from the variable property analysis for the cases considered (i.e., step 1), and the values of  $\theta'(0)_{c.p.}$  were calculated from Equation (7.8).

5. Development of a general plot for reference temperature. The reference temperature for fluids in the supercritical region is a

complicated function of wall, free stream, and transposed critical temperatures. Therefore, it was not possible to express this temperature with a single equation that would represent the fluids in the supercritical region under the physical conditions considered. The next best possible approach in presentation of reference temperature was to express the variation of reference temperature  $T_x$  with wall temperature and ratio of wall to free stream temperature. The plot of  $x = (T_w - T_x)/(T_w - T_\infty)$  versus  $T_w/T_\infty$  for various wall temperatures is shown in Figure 18 (Appendix B). This plot was generated for Refrigerant-114 at a constant pressure of 540 psia, and three different wall temperatures.

In an attempt to obtain a presentation for reference temperature which is more easily interpretable, the parameter  $x$  (expressing the ratio of the wall temperature  $T_w$  minus reference temperature  $T_x$  to wall temperature minus free stream temperature  $T_\infty$ ) was plotted over the parameter  $(T_M - T_\infty)/(T_w - T_\infty)$  in Figure 19 (Appendix B). In this parameter,  $T_M$  is that temperature at which specific heat assumes its maximum value. It may be observed that in this presentation the influence of wall temperature  $T_w$  has been eliminated, since all the curves for different  $T_w$  values align quite well along a single curve.

Steps 1 through 5 presented above complete the analysis and development of the reference temperature method. The results obtained based on these steps were then applied to several fluids for wide ranges of supercritical pressures and temperatures. These results will be presented and discussed next.

As presented in Figure 19, the influence of wall temperature can be eliminated if  $x = (T_w - T_x)/(T_w - T_\infty)$  is represented versus  $(T_M - T_\infty)/(T_w - T_\infty)$  rather than  $T_w/T_\infty$  (see Figure 18). The next step in the

development of the reference temperature analysis was to generalize the single curve of Figure 19 to cover a wide range of supercritical temperatures and pressures for several fluids. For this purpose the three fluids, Refrigerant-114, water, and carbon dioxide, were chosen. The ranges of supercritical temperatures and pressures used to develop the reference temperature plots for the three fluids were:

1. Refrigerant-114
  - a. Pressure range,  $P = 500, 540, 600$  psia
  - b. Temperature range,  $T = 300$  to  $350^{\circ}\text{F}$
2. Water
  - a. Pressure range,  $P = 3400, 3600, 3800$  psia
  - b. Temperature range,  $T = 715$  to  $750^{\circ}\text{F}$
3. Carbon Dioxide
  - a. Pressure range,  $P = 1200, 1400, 1600$  psia
  - b. Temperature range,  $T = 90$  to  $140^{\circ}\text{F}$ .

The ranges of pressures and temperatures chosen for the above fluids cover the experimental pressure and temperature ranges reported in the literature (see Table I, Appendix A).

Considering the five major steps outlined for the development of the reference temperature, the first step in generalization of Figure 19 was to obtain the dimensionless heat transfer coefficients. For each one of the pressures assumed for the three fluids, as given above, ten different wall temperatures were chosen. These temperatures were selected in such a way that they would cover the wide range of temperatures encountered for each fluid, as given above. For each case the wall temperature was kept constant and the free stream temperature was varied to within one degree of the specified wall temperature. The

dimensionless heat transfer coefficients for these problems relating to the supercritical region were then calculated based on the equations, procedures, and computer programs introduced in Chapters III, IV, and V. The variable property heat transfer results for each fluid for the range of temperatures and pressures employed were then correlated with the same type of correlation given by Equation (7.5). The constants  $a$ ,  $b$ ,  $c$ , and  $d$  of Equation (7.5) obtained for the three fluids are tabulated in Table IX (Appendix A). This table also lists the percent deviation between the correlated and calculated results, and the range of pressures and temperatures used for these correlations. The constants of Equation (7.5), tabulated in Table IX were obtained from the MARQ program.

The reference temperature results for Refrigerant-114, water, and carbon dioxide are presented in Figures 20, 21, and 22 (Appendix B), respectively. These figures cover a wide range of supercritical temperatures and pressures for each fluid. Figures 20, 21, and 22 represent the variation of  $x = (T_w - T_x)/(T_w - T_\infty)$  with  $(T_M - T_\infty)/(T_w - T_\infty)$  for various wall and free stream temperatures at three specific pressures for each fluid. These figures were produced following the five-step procedure outlined earlier for the development of reference temperature.

Before discussing the results of Figures 20, 21, and 22, it would be important to first concentrate on an isolated case, and then consider the more generalized results. For this purpose the reference temperature results of Refrigerant-114 at a constant pressure of 540 psia were chosen (see Figure 20). These results are represented in Figure 23 (Appendix B) in terms of variation of  $x = (T_w - T_x)/(T_w - T_\infty)$  with  $(T_M - T_\infty)/(T_w - T_\infty)$  at various wall temperatures for a constant pressure.

As shown in Figure 23 the data points align quite well along a single curve.

There are several interesting points to be noted about the single curve of Figure 23. The curve of  $x = (T_w - T_x)/(T_w - T_\infty)$  behaves very nicely and smoothly up to certain values of  $(T_M - T_\infty)/(T_w - T_\infty)$  and then there is a sudden abrupt change in the direction and behavior of the curve. This abrupt change in the direction of the curve is primarily due to the large influence of  $T_M$ , the temperature at which specific heat assumes its maximum, on the reference temperature. This change occurs for the values of  $(T_M - T_\infty)/(T_w - T_\infty)$  in the range of -0.25 to 2.00. These values of  $(T_M - T_\infty)/(T_w - T_\infty)$  occur where the wall temperature  $T_w$  or the free stream temperature  $T_\infty$  are close to  $T_M$  ( $T_M = 306^\circ\text{F}$  for Refrigerant-114 at a pressure of 540 psia), that is, when  $T_w$  or  $T_\infty$  are within about five degrees F of  $T_M$  on both sides of the specific heat peak. Figures 24 and 25 (Appendix B) show the variations in the thermodynamic and transport properties of Refrigerant-114 with respect to temperature at a constant pressure of 540 psia. From these figures it can be seen that around the temperature of  $T_M = 306^\circ\text{F}$  the physical properties that go through an abrupt change are the specific heat and Prandtl number. This abrupt change in the specific heat has a direct and strong influence on the reference temperature.

For the cases where both the wall temperature and free stream temperature are away from the  $T_M$  (for the results of Figure 23 this would be on the right side of the specific heat maximum) the direct and strong influence of  $T_M$  on the reference temperature vanishes and the variation of  $x = (T_w - T_x)/(T_w - T_\infty)$  with  $(T_M - T_\infty)/(T_w - T_\infty)$  can be represented by a simple curve. For the results presented in Figure 23, this simple



curve covers the values of  $(T_M - T_\infty)/(T_W - T_\infty)$  in the range of -43.0 to -0.25.

As shown in Figures 20, 21, and 22 the variation of  $x = (T_W - T_x)/(T_W - T_\infty)$  versus  $(T_M - T_\infty)/(T_W - T_\infty)$  for Refrigerant-114, water, and carbon dioxide at different supercritical pressures and wide range of temperatures show results similar to the ones presented in Figure 23 for Refrigerant-114 at a constant pressure of 540 psia. Therefore, the very same observations and conclusions made about the results of Figure 23, which were for a specific fluid at a single pressure, can be employed for the results of Figures 20, 21, and 22, which are for several fluids at several different pressures and temperatures.

Figures 20, 21, and 22 indicate that it is possible to obtain a single curve for different pressures up to a point where the direct influence of  $T_M$  on the reference temperature is not significant, that is, for the cases where the wall and free stream temperatures were more than five degrees to the right of the specific heat peak. For the cases where  $T_W$  or  $T_\infty$  are in the vicinity of  $T_M$ , a single curve was not obtainable. This was primarily due to the fact that for each pressure the specific heat assumes a different maximum value, since in these cases  $T_M$  has a direct and significant influence on the prediction of the reference temperature.

However, it is interesting to note that in Figures 20, 21, and 22, for the cases where  $T_W$  or  $T_\infty$  were close to  $T_M$ , the reference temperature curves for all three pressures considered for each fluid branched off from the same line. The amount of departure of these curves from the single curve increases as the pressure is increased, since the higher the pressure the further is the maximum peak of specific heat removed

from the critical point. For the cases where the free stream temperature was on the left of the specific heat peak, that is where the ratio  $(T_M - T_\infty)/(T_W - T_\infty)$  was positive; as soon as the direct influence of the specific heat maximum on the reference temperature was diminished, the reference temperature curves behaved smoothly again. This phenomenon can best be observed by considering those reference temperature curves presented in Figures 20, 21, and 22, which were obtained for the highest pressures utilized.

The solid lines drawn through the data points given in Figures 20, 21, and 22 present the best fit obtained for the reference temperature data generated for the fluids under study. Plots similar to the one given in Figure 23 for Refrigerant-114 at a constant pressure of 540 psia were generated for the three fluids at every one of the pressures considered. Then, the best fit through the data points in these plots was demonstrated by a solid curve which expresses the variation of  $x = (T_W - T_x)/(T_W - T_\infty)$  versus  $(T_M - T_\infty)/(T_W - T_\infty)$ . These results, as mentioned earlier, are presented in Figures 20, 21, and 22.

In order to complete the analysis of the reference temperature method, it was necessary to establish the validity and universality of this method. This was accomplished by comparing the constant property heat transfer results calculated with the predicted reference temperatures obtained from Figures 20, 21, and 22 with the experimental and theoretical variable property heat transfer results presented in Chapter VI for the three fluids under study. In addition, some of the variable property heat transfer results obtained with the use of the equations, procedures, and computer programs introduced in Chapters III, IV,

and  $V$  were compared with the constant property heat transfer results obtained with the predicted reference temperatures.

In order to calculate the constant property heat transfer results, for the free convective heat transfer problems considered in these comparisons, the dimensionless reference temperatures  $x = (T_w - T_x) / (T_w - T_\infty)$ , were obtained from Figures 20, 21, or 22 for different values of  $(T_M - T_\infty) / (T_w - T_\infty)$  for the fluids of interest. The reference temperatures were then calculated using the following equation:

$$T_{\text{ref}} = T_x = T_w - x(T_w - T_\infty) \quad (7.11)$$

These reference temperatures were used to evaluate the value of Prandtl number in the constant property equations. The following constant property equations for momentum and energy were used:

$$F''' = -3FF'' + 2(F')^2 - \theta \quad (7.12)$$

$$\theta'' = -3Pr_{\text{ref}} F\theta' \quad (7.13)$$

where  $Pr_{\text{ref}}$  is Prandtl number evaluated at a reference temperature.

The boundary conditions for Equations (7.12) and (7.13) are:

$$\left. \begin{array}{l} F = 0 \\ F' = 0 \\ \theta = 1 \end{array} \right\} \eta = 0 \quad \left. \begin{array}{l} F' = 0 \\ \theta = 0 \end{array} \right\} \eta = \infty \quad (7.14)$$

Equations (7.12) and (7.13) along with the boundary conditions given by Equation (7.14) were solved on the computer using the Lentini-Pereyra Program (see Chapter V). The slope of the dimensionless temperature profile at the wall,  $\theta'(0)$  obtained from the solution of the conservation equations was used in the following equations to obtain the local heat flux and the local dimensionless heat transfer coefficient

(local Nusselt number) for the constant property problem:

$$(\dot{q}_{x,w})_{c.p.} = -K_{ref}(T_w - T_\infty) C_{ref} x^{-1/4} [\theta'(0)]_{ref} \quad (7.15)$$

where

$$C_{ref} = \left[ \frac{g \rho_{ref}^2}{4 \mu_{ref}^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right) \right]^{1/4}$$

and

$$(\text{Nu}_{x,w})_{c.p.} = - \frac{\text{Gr}_{x,ref}^{1/4}}{\sqrt{x}} [\theta'(0)]_{ref} \quad (7.16)$$

where

$$\text{Gr}_{x,ref} = \left[ \frac{g x^3 \rho_{ref}^2}{\mu_{ref}^2} \left( \frac{\rho_\infty - \rho_w}{\rho_w} \right) \right]$$

In Equations (7.15) and (7.16) the subscript (c.p.) designates the constant property results.

The constant property heat flux results obtained from Equation (7.15) were compared with the theoretical and experimental results presented in Chapter V. This comparison is shown in Figure 26 (Appendix B). From this figure it can be seen that the constant property heat flux results predicted by the reference temperature method are within an error of  $\pm 20$  percent of the variable property experimental and theoretical results.

In addition to the above comparison, the variable property heat transfer results of Refrigerant-114 at 540 psia, based on the equations, procedures, and computer programs introduced in Chapters III, IV, and V were compared with the constant property heat transfer results obtained

with the predicted reference temperatures. This was achieved by choosing 37 points on the 540 psia curve of Figure 20 for Refrigerant-114 that would best reproduce the curve. Most of the data points were chosen for the wall and free stream temperatures that best represented the abrupt change in the curve. The heat transfer results obtained for these data points cover a wide temperature range of 300 to 350°F. The comparison between the constant property Nusselt numbers obtained from Equation (7.16) and the variable property Nusselt numbers obtained from Equation (3.26) are presented in Figure 27 (Appendix B). From this figure it can be seen that the constant property Nusselt numbers predicted by the reference temperature method are within an error of  $\pm 10$  percent of the variable property results.

In general, based on the results presented in this chapter for the reference temperature method, it can be concluded that one single curve for prediction of reference temperature in certain regions can be established. That is, the region where the influence of  $T_M$ , the temperature at which specific heat assumes its maximum value, is not directly significant on the reference temperature. The heat transfer results obtained using the predicted reference temperatures in conjunction with the constant property equations when compared with the experimental and theoretical variable property heat transfer results of Refrigerant-114, water, and carbon dioxide, produced very good results. For the regions where the influence of  $T_M$  on the reference temperature was significant, it was still possible to predict reference temperature and in turn heat transfer coefficients with good accuracy, but the curves used for these predictions were pressure dependent.

As pointed out earlier the utility of the reference temperature lies in the fact that it allows constant property results to be used to compute variable property results. Once the reference temperature for a particular problem under study is specified, it is an easy matter to obtain the variable property heat transfer results. This can be accomplished by obtaining the local heat flux or local Nusselt number from Equations (7.15) or (7.16), respectively. The requirements for evaluation of these equations are the physical property information, and the constant property solutions, that is, the slope of the temperature profile at the wall as a function of the Prandtl number evaluated at the specified reference temperature. The physical property information can be obtained from the equations and computer programs developed in Chapters IV and V, respectively. The solution to the constant property problem can be acquired from the solution of the conservation of momentum and energy equations, Equations (7.12) and (7.13), along with their boundary conditions, Equation (7.14), with the use of the Lentini-Pereyra Program (see Chapter V). To simplify the reference temperature method one step further, the solution to the constant property equations for the range of Prandtl numbers considered in this study are presented in Figure 28 (Appendix B). These constant property results were also correlated as a function of Prandtl number according to the following equation:

$$-\theta'(0) = a_1 + b_1 (\text{Pr})^{c_1} \quad (7.17)$$

where

$$a_1 = 0.10;$$

$$b_1 = 0.48615754; \text{ and}$$

$$c_1 = 0.33141328.$$

The constants  $a_1$ ,  $b_1$ , and  $c_1$  of Equation (7.17) were obtained using the MARQ program. Equation (7.17) with the constants given above predicts the solution to the constant property problems within an average absolute error of 1.3 percent. The above equation covers a range of Prandtl number from 0 to 40.

It should also be pointed out that for a specified problem the free convective heat transfer coefficients to fluids in the supercritical region with the exception of liquid metals, can be directly calculated from the following equation:

$$Nu_{x,w} = \frac{h_x x}{K_w} = a(Gr_{x,w})^b (Pr_\infty)^c \left(\frac{T_\infty}{T_w - T_\infty}\right)^d \quad (7.18)$$

The constants  $a$ ,  $b$ ,  $c$ , and  $d$  of Equation (7.18) obtained for Refrigerant-114, water, and carbon dioxide along with its range of application are tabulated in Table IX (Appendix A). Equation (7.18) requires information on physical properties, which can be obtained from the equations and computer programs developed in Chapters IV and V, respectively.

## CHAPTER VIII

### SUMMARY, CONCLUSIONS, AND RECOMMENDATIONS

#### Summary and Conclusions

The accomplishments of this investigation may be summarized as follows:

1. The fundamental partial differential equations of conservation of mass, momentum, and energy for laminar free convection in a boundary layer along a vertical flat plate, Equations (3.1) through (3.3), were reduced to a set of two ordinary differential equations, Equations (3.14) and (3.15), by means of the compressible stream function and a similarity variable, Equations (3.5) and (3.6). In the development of these reduced differential equations no simplifying assumptions were made regarding the physical properties. The property coefficients, Equations (3.16) through (3.20), were expressed as functions of  $\theta(\eta)$ , once expressions for the physical properties  $\rho$ ,  $C_p$ ,  $\mu$ , and  $K$  as a function of temperature at a particular pressure are known.

2. Expressions for thermodynamic and transport properties in the supercritical region as functions of pressure and temperature were developed for Refrigerant-114, water, and carbon dioxide (see Chapter IV). Development of these expressions was necessary in order to solve the reduced ordinary differential momentum and energy equations, Equations (3.14) and (3.15), for the variable property case.



3. The numerical solution of the reduced differential momentum and energy equations is complicated by the fact that two of the five wall boundary conditions are unknown. These two unknown boundary conditions are  $F''(0)$  and  $\theta'(0)$  and their values must be obtained in order to evaluate the heat transfer and shearing stress. The iterative finite difference numerical technique of Lentini and Pereyra (8) which was described in Chapter V was demonstrated to be a very effective and practical numerical method for overcoming this difficulty. This numerical procedure is easy to use, efficient, accurate, and general.

4. The validity and universality of the present variable property free convective heat transfer model has been demonstrated in Chapter VI for three fluids at supercritical conditions. The present prediction scheme showed very good agreement with the theoretical heat transfer results of Parker and Mullin (5) for Refrigerant-114, and experimental heat transfer results of Fritsch and Grosh (12) and Nishikawa et al. (30) for water and carbon dioxide, respectively (see Figures 10, 11, 14, and 15).

5. The present model also proved to be more general (i.e., it is independent of the fluid of interest), less complicated, easier to use, and more accurate in comparison to the specific models of Nowak and Konanur (7) for water, Nishikawa et al. (6) for carbon dioxide, and Parker and Mullin (5) for Refrigerant-114 (see Figures 10, 13, and 14).

6. The validity of the present model in predicting the constant property results was also presented in Chapter VI. The prediction scheme showed excellent agreement with the well-known constant property analysis of Ostrach (10) (see Figures 16 and 17).

7. In Chapter VII a shorthand method was developed by which heat transfer to fluids in the supercritical region under variable property conditions in laminar free convection on a vertical flat plate can be evaluated. This technique is referred to as the reference temperature method. Based on this method three generalized plots for Refrigerant-114, water, and carbon dioxide were developed. These plots cover a wide range of supercritical temperatures and pressures (see Figures 20, 21, and 22). Based on the results of these plots, it can be concluded that one single curve for prediction of reference temperature in certain regions can be established. That is, the region where the influence of  $T_M$ , the temperature at which specific heat assumes its maximum value, is not directly significant on the reference temperature. For the regions where the influence of  $T_M$  on the reference temperature was significant it was still possible to predict reference temperature, but the curves used for these predictions were pressure dependent.

8. The utility of the reference temperature lies in the fact that it allows constant property results to be used to compute variable property results. Once the reference temperature for a particular problem under study is specified, it is an easy matter to obtain the variable property heat transfer results. This can be accomplished by obtaining the local heat flux or local Nusselt number from Equations (7.15) or (7.16), respectively. The requirements for evaluation of these equations are the physical property information, and the constant property solutions, that is, the slope of the temperature profile at the wall as a function of the Prandtl number evaluated at the specified reference temperature.

9. The validity and universality of the reference temperature method has been demonstrated in Chapter VII for the three fluids. The heat transfer results obtained with the reference temperature scheme showed very good agreement with the theoretical heat transfer results of Parker and Mullin (5) for Refrigerant-114, and experimental heat transfer results of Fritsch and Grosh (12) and Nishikawa et al. (30) for water and carbon dioxide, respectively (see Figure 26).

10. The validity of the reference temperature method was also checked against the theoretical heat transfer results of Refrigerant-114 at 540 psia obtained from the variable property analysis and Lentini-Pereyra Program (see Chapters III, IV, and V). These heat transfer data covered a wide range of temperatures which were not encountered with the available experimental data. The comparison between the theoretical variable property results and the constant property results obtained in conjunction with the predicted reference temperatures showed excellent agreement (see Figure 27).

11. In general, based on the results presented in Chapter VII for the reference temperature method, it can be concluded that the heat transfer results obtained using the predicted reference temperature in conjunction with the constant property equations when compared with the experimental and theoretical variable property heat transfer results of Refrigerant-114, water, and carbon dioxide produced very good results.

12. For the three fluids encountered in this study a correlation was also developed which would predict heat transfer once the physical properties for the problem under study are known (see Equation (7.18)). This correlation is applicable to Refrigerant-114, water, and carbon dioxide in the range of temperatures and pressures considered (see Table IX).

## Recommendations

Based on the observations made during this study, the following recommendations are made:

1. The variable property prediction scheme should be investigated with more heat transfer experimental results which cover wide ranges of supercritical pressures and temperatures. Such experimental results are not available.
2. Accurate data on density, specific heat, absolute viscosity, and thermal conductivity in the critical and supercritical regions are needed for different fluids.
3. Since the physical properties in the supercritical region vary peculiarly with respect to temperature at a constant pressure, it is essential that the experimental investigations for the physical properties should be carried out at small temperature increments to insure accurate determination of these variations.

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APPENDIX A

TABLES

TABLE I  
SUMMARY OF FREE CONVECTION EXPERIMENTAL WORK

Investigator	Date of Work	Reference	Fluid	Geometry	Range of Conditions	
					Ratio of Pressure to Critical Pressure $P/P_c$	Ratio of Temperature to Critical Temperature $T/T_c$
Doughty and Drake	1956	16	R-12 ( $CCl_2F_2$ )	Horizontal Wire	0.28 to 1.70	0.77 to 1.04
Bonilla and Sigel	1960	17	n-pentane ( $C_5H_{12}$ )	Horizontal Plate	0.72 to 1.50	0.65 to 0.74 (fluid) 0.71 to 1.06 (surface)
Griffith and Sabersky	1960	18	R-114 ( $C_2Cl_2F_4$ )	Horizontal Wire	0.42 to 1.15	0.725 to 1.080
Skipov and Potashev	1962	19	$CO_2$	Vertical Wire	0.82 to 1.25	1.00 to 1.02
Brodowicz and Bialokoz	1963	20	R-12 ( $CCl_2F_2$ )	Vertical Plate	1	0.98 to 1.07
Holt and Grosh	1963	21	$H_2O$	Horizontal Wire and Vertical Ribbon	$1.5 \times 10^{-4}$ to 1.25	0.75 to 1.00 (fluid) 0.75 to 2.00 (surface)

TABLE I (Continued)

Investigator	Date of Work	Reference	Fluid	Geometry	Range of Conditions	
					Ratio of Pressure to Critical Pressure $P/P_c$	Ratio of Temperature to Critical Temperature $T/T_c$
Fritsch and Grosh	1963	12	H <sub>2</sub> O	Vertical Ribbon	1.03 to 1.06	1.0036 to 1.0086
Dubrovina and Skripov	1964	22	CO <sub>2</sub>	Horizontal Wire, Horizontal and Vertical Ribbon	0.99 to 1.30	1.00 to 1.03
Knapp and Sabersky	1965	23	CO <sub>2</sub>	Horizontal and Vertical Wire	1.03 to 1.40	0.93 to 1.09
Nishikawa and Miyabe	1965	24	CO <sub>2</sub>	Horizontal Wire	1.00 to 1.33	0.98 to 1.63
Daniels and Bramall	1965	25	CO <sub>2</sub>	Horizontal Monel Wire	0.79 to 1.16	0.86 to 1.03
Graham et al.	1965	26	H <sub>2</sub>	Horizontal Ribbon	0.32 to 1.38	0.76 to 1.19
Larson and Schoenhals	1966	27	H <sub>2</sub> O	Vertical Ribbon	1.01 to 1.03	0.98 to 1.04
Goldstein and Aung	1967	28	CO <sub>2</sub>	Horizontal Wire	1.03 to 1.21	0.94 to 1.09

TABLE I (Continued)

Investigator	Date of Work	Reference	Fluid	Geometry	Range of Conditions	
					Ratio of Pressure to Critical Pressure $P/P_c$	Ratio of Temperature to Critical Temperature $T/T_c$
Kato et al.	1967	29	CO <sub>2</sub>	Vertical Wire	1.097	0.96 to 1.54
Nishikawa et al.	1967	30	CO <sub>2</sub>	Vertical Wire	1.097	0.96 to 1.74

TABLE II  
SOURCES OF PHYSICAL PROPERTIES FOR FLUIDS IN THE SUPERCRITICAL REGION

Fluid	Critical Constants			Source Reference		Properties Recommended as Computable by Source Authors							
	Pressure, $P_c$ , atm	Temperature, $T_c$ , °K	Density, $\rho_c$ , g/cm <sup>3</sup>	Equation of State	Transport Properties	Pressure- Density- Temperature P- $\rho$ -T	En- thalpy H	Specific Heat at Constant Pressure $C_p$	Specific Heat at Constant Volume $C_v$	Expansion Coefficient $\beta = -\frac{1}{\rho} \left( \frac{\partial \rho}{\partial T} \right)_P$	En- tropy S	Abs- olute Viscos- ity $\mu$	Thermal Conduc- tivity K
Helium-4	2.26	5.30	0.0692	31	37,38	x	x				x	x	x
Hydrogen	12.80	32.99	0.0314	32	37,38	x	x	x	x	x	x	x	x
Nitrogen	33.50	126.20	0.3110	33	37,38	x	x				x	x	x
Oxygen	50.14	154.80	0.4358	34	37,38	x	x	x	x	x	x	x	x
Propane	42.10	370.00	0.2100	35	---	x	x	x	x	x	x		
Water	218.20	647.20	0.3220	*	37-41	x	x	x	x	x	x	x	x
Carbon Dioxide	72.90	304.20	0.4690	*	42-49	x	x	x	x	x		x	x
R-12	40.69	385.10	0.5570	36	---	x	x		x		x		
R-114	31.45	418.70	0.5820	13	50	x		x	x	x		x	x
Ammonia	112.30	406.10	0.2360	---	37,38	x				x		x	x

\*Present study.

TABLE III  
TABLE OF  $\bar{C}$  VALUES FOR GAS A

$\frac{T_w}{T_\infty}$	$-\frac{\theta'(0)_w}{\sqrt{2}}$	$\frac{\theta'(0)_{ref}^*}{\theta'(0)_w}$	$\left(\frac{\theta'(0)_{ref}}{\theta'(0)_w}\right)^8$	$1 - \left(\frac{\theta'(0)_{ref}}{\theta'(0)_w}\right)^8$	$\frac{T_\infty}{T_w}$	$1 - \left(\frac{T_\infty}{T_w}\right)$	$\bar{C}$
4	0.371	0.95148	0.672	0.328	1/4	3/4	0.438
3	0.368	0.95920	0.716	0.284	1/3	2/3	0.427
5/2	0.366	0.96448	0.749	0.251	2/5	3/5	0.418
2	0.363	0.97245	0.799	0.201	1/2	1/2	0.402
3/4	0.348	1.01436	1.122	-0.122	4/3	-1/3	0.366
1/2	0.339	1.04129	1.383	-0.383	2/1	-1	0.383
1/3	0.330	1.06969	1.714	-0.714	3/1	-2	0.357
1/4	0.323	1.09287	2.035	-1.035	4/1	-3	0.345

\*Where  $\theta'(0)_{ref}$  was obtained from constant property solution with  $Pr = 0.7$ .

$$Nu_{x,w}/Gr_{x,w}^{1/4} = -(\theta'(0)_{ref}/\sqrt{2}) = 0.353.$$



TABLE IV  
UNITS AND CONSTANTS FOR EQUATION OF STATE

Parameter	Description	Units	Refrigerant-114	Water	Carbon Dioxide
P	Pressure	lbf/in. <sup>2</sup>	---	---	---
T	Temperature	°R	---	---	---
V	Specific Volume	ft <sup>3</sup> /lbm	---	---	---
T <sub>c</sub>	Critical Temperature	°R	753.95	1165.30	547.56
R	Gas Constant	ft <sup>3</sup> /°R-in. <sup>2</sup>	0.0628018	0.59569248	0.24384004
K	Constant	---	3.00	0.86919801	0.12018118 x 10 <sup>1</sup>
b	Constant	---	5.9149070 x 10 <sup>-3</sup>	-0.24670554 x 10 <sup>-1</sup>	-0.18588948 x 10 <sup>-1</sup>
A <sub>2</sub>	Constant	---	-2.3856704	-0.22524809 x 10 <sup>2</sup>	-0.13326704 x 10 <sup>2</sup>
B <sub>2</sub>	Constant	---	1.0801207 x 10 <sup>-3</sup>	0.11652976 x 10 <sup>-1</sup>	-0.83298325 x 10 <sup>-2</sup>
C <sub>2</sub>	Constant	---	-6.5643648	-0.70259944 x 10 <sup>2</sup>	0.62417313 x 10 <sup>2</sup>
A <sub>3</sub>	Constant	---	3.4055687 x 10 <sup>-2</sup>	-0.87423115 x 10 <sup>1</sup>	-0.73090008 x 10 <sup>1</sup>
B <sub>3</sub>	Constant	---	-5.3336494 x 10 <sup>-6</sup>	0.70670168 x 10 <sup>-2</sup>	0.90982283 x 10 <sup>-2</sup>
C <sub>3</sub>	Constant	---	1.6366057 x 10 <sup>-1</sup>	-0.19906674	0.57368866 x 10 <sup>1</sup>
A <sub>4</sub>	Constant	---	-3.8574810 x 10 <sup>-4</sup>	0.71943958 x 10 <sup>-1</sup>	0.18744415 x 10 <sup>-1</sup>
A <sub>5</sub>	Constant	---	1.6017659 x 10 <sup>-6</sup>	0.12179538 x 10 <sup>-1</sup>	0.10958040 x 10 <sup>-1</sup>
B <sub>5</sub>	Constant	---	6.2632340 x 10 <sup>-10</sup>	-0.15583440 x 10 <sup>-4</sup>	-0.12655010 x 10 <sup>-4</sup>
C <sub>5</sub>	Constant	---	-1.0165314 x 10 <sup>-5</sup>	0.13679639 x 10 <sup>-1</sup>	-0.13531694 x 10 <sup>-1</sup>

TABLE V

## UNITS AND CONSTANTS FOR SPECIFIC HEAT AT CONSTANT PRESSURE

Parameter	Description	Units	Refrigerant-114	Water	Carbon Dioxide
$C_p$ or $C_v$	Specific Heat	Btu/lbm-°R	---	---	---
T	Temperature	°R	---	---	---
V	Specific Volume	ft <sup>3</sup> /lbm	---	---	---
$T_c$	Critical Temperature	°R	753.95	1165.30	547.56
R	Gas Constant	ft <sup>3</sup> /°R-in. <sup>2</sup>	0.0628018	0.59569248	0.24384004
K	Constant	---	3.0	0.84750477	$0.12092998 \times 10^1$
b	Constant	---	$5.9149070 \times 10^{-3}$	$-0.23690403 \times 10^{-1}$	$-0.30146832 \times 10^{-1}$
$A_2$	Constant	---	-2.3856704	$-0.22981253 \times 10^2$	$-0.10972492 \times 10^2$
$B_2$	Constant	---	$1.0801207 \times 10^{-3}$	$0.11977436 \times 10^{-1}$	$-0.56307166 \times 10^{-2}$
$C_2$	Constant	---	-6.5643648	$-0.72422920 \times 10^2$	$0.47571869 \times 10^2$
$A_3$	Constant	---	$3.4055687 \times 10^{-2}$	$-0.87176941 \times 10^1$	$-0.72139233 \times 10^1$
$B_3$	Constant	---	$-5.3336494 \times 10^{-6}$	$0.71550030 \times 10^{-2}$	$0.90332088 \times 10^{-2}$
$C_3$	Constant	---	$1.6366057 \times 10^{-1}$	-0.19803821	$0.63052626 \times 10^1$
$A_4$	Constant	---	$-3.8574810 \times 10^{-4}$	$0.77529884 \times 10^{-1}$	$0.13941827 \times 10^{-2}$
$A_5$	Constant	---	$1.6017659 \times 10^{-6}$	$0.11963517 \times 10^{-1}$	$0.11295915 \times 10^{-1}$
$B_5$	Constant	---	$6.2632340 \times 10^{-10}$	$-0.15597259 \times 10^{-4}$	$-0.12727667 \times 10^{-4}$
$C_5$	Constant	---	$-1.0165314 \times 10^{-5}$	$0.12926977 \times 10^{-1}$	$-0.13171462 \times 10^{-1}$

TABLE V (Continued)

Parameter	Description	Units	Refrigerant-114	Water	Carbon Dioxide
$X_{13}$	Constant	---	0.0175	$0.11488819 \times 10^1$	$0.70600271 \times 10^1$
$X_{14}$	Constant	---	$3.49 \times 10^{-4}$	$0.44963023 \times 10^{-4}$	$-0.17494759 \times 10^{-1}$
$X_{15}$	Constant	---	$-1.67 \times 10^{-7}$	$0.28090134 \times 10^{-8}$	$0.10779554 \times 10^{-4}$

TABLE VI  
UNITS AND CONSTANTS FOR ABSOLUTE VISCOSITY

Parameter	Description	Units	Refrigerant-114	Water	Carbon Dioxide
$\mu$	Viscosity	Centipoises* or lbm/ft-sec	---	---	---
$\mu_0$	Low Pressure Viscosity	Centipoises	---	---	---
M	Molecular Weight	---	170.936	---	---
$P_c$	Critical Pressure	Atmosphere	32.190	---	---
$T_c$	Critical Temperature	$^{\circ}\text{K}$	418.880	---	---
T	Temperature	$^{\circ}\text{R}$	---	---	---
$\rho_c$	Critical Density	$\text{g/cm}^3$	0.582	---	---
$\rho$	Density	$\text{g/cm}^3$ * $\text{lbm/ft}^3$ or	---	---	---
A	Constant	---	---	$0.9717059 \times 10^{-5}$	$-0.2003388 \times 10^{-5}$
B	Constant	---	---	$0.6335240 \times 10^{-8}$	$0.2267791 \times 10^{-7}$
C	Constant	---	---	$0.1024267 \times 10^{-6}$	$0.1030594 \times 10^{-7}$
D	Constant	---	---	1.57	$0.2089048 \times 10^1$

Note: The units designated by (\*) exclusively refer to Refrigerant-114.

TABLE VII  
UNITS AND CONSTANTS FOR THERMAL CONDUCTIVITY

Parameter	Description	Units	Refrigerant-114	Water	Carbon Dioxide
K	Thermal Conductivity	$\frac{\text{cal}^*}{\text{sec-cm-}^\circ\text{K}}$ or $\frac{\text{Btu}}{\text{hr-ft-}^\circ\text{F}}$	---	---	---
$K_0$	Low Pressure Thermal Conductivity	cal/sec-cm- $^\circ\text{K}$	---	---	---
$\mu_0$	Low Pressure Viscosity	centipoises	---	---	---
$C_{V_0}$	Low Pressure Specific Heat	cal/g-mole- $^\circ\text{C}$	---	---	---
M	Molecular Weight	---	170.936	---	---
R	Gas Constant	cal/g-mole- $^\circ\text{K}$	1.987	---	---
$P_C$	Critical Pressure	atmosphere	32.190	---	---
$T_C$	Critical Temperature	$^\circ\text{K}$	418.880	---	---
T	Temperature	$^\circ\text{R}$ or $^\circ\text{F}^+$	---	---	---
$\rho_C$	Critical Density	g-mole/cm <sup>3</sup>	99.485	---	---
$\rho$	Density	$\frac{\text{g-mole}^*}{\text{cm}^3}$ or $\frac{\text{lbm}}{\text{ft}^3}$	---	---	---
A	Constant	---	---	$3.6200 \times 10^{-3}$	$0.84746444 \times 10^{-2}$
B	Constant	---	---	$0.0380 \times 10^{-3}$	$-0.44669005 \times 10^{-4}$
C	Constant	---	---	$2.6670 \times 10^{-3}$	$0.11056622 \times 10^{-1}$
D	Constant	---	---	1.24	0.46278907

Note: The units designated by (\*) or (+) exclusively refer to Refrigerant-114 and water, respectively.

TABLE VIII  
SUMMARY OF PHYSICAL PROPERTY EQUATIONS

Fluid	Eq. No. (See Ch. IV)	Source Ref.	No. of Points	Percent Deviation	Eq. No. (See Ch. IV)	Source Ref.	No. of Points	Percent Deviation	$T_r = T/T_c$ $P_r = P/P_c$	
	Equation of State				Specific Heat at Constant Pressure				Range of Application	
Water	(4.1)	40 41 53	64	<u>+1.38</u>	(4.12)	37,38, 40,41, 59	64	<u>+3.90</u>	1.0080 to 1.0400	1.0440 to 1.2470
Carbon Dioxide	(4.1)	54-56	52	<u>+1.80</u>	(4.12)	46-49	34	<u>+3.80</u>	1.0028 to 1.1455	1.0560 to 1.9550
Refrigerant -114	(4.1)	13	---	---	(4.12)	50	---	---	1.0076 to 1.1461	1.0355 to 2.3754
Fluid	Absolute Viscosity				Thermal Conductivity				Range of Application	
Water	(4.15)	39-41	64	<u>+0.53</u>	(4.26)	39-41	64	<u>+2.10</u>	1.0080 to 1.0400	1.0440 to 1.2470
Carbon Dioxide	(4.15)	42,43	72	<u>+1.40</u>	(4.26)	44,45	39	<u>+2.50</u>	1.0028 to 1.1455	1.0560 to 1.9550
Refrigerant -114	(4.13) and (4.14)	50	---	---	(4.24) and (4.25)	50	---	---	1.0076 to 1.1461	1.0355 to 2.3754

TABLE IX  
CONSTANTS FOR HEAT TRANSFER CORRELATION

Fluid	Constants				Percent Deviation	Range of Application	
	a	b	c	d		$T_r = T/T_c$	$P_r = P/P_c$
Refrigerant -114	0.39992773	0.25	0.30848627	0.0049719059	1.5	1.0076 to 1.0740	1.0638 to 1.2766
Water	0.31530544	0.25	0.39214071	-0.022304567	2.5	1.0080 to 1.0380	1.0598 to 1.1845
Carbon Dioxide	0.32646432	0.25	0.40539598	-0.027499479	2.4	1.0038 to 1.0950	1.1194 to 1.4925

APPENDIX B

FIGURES



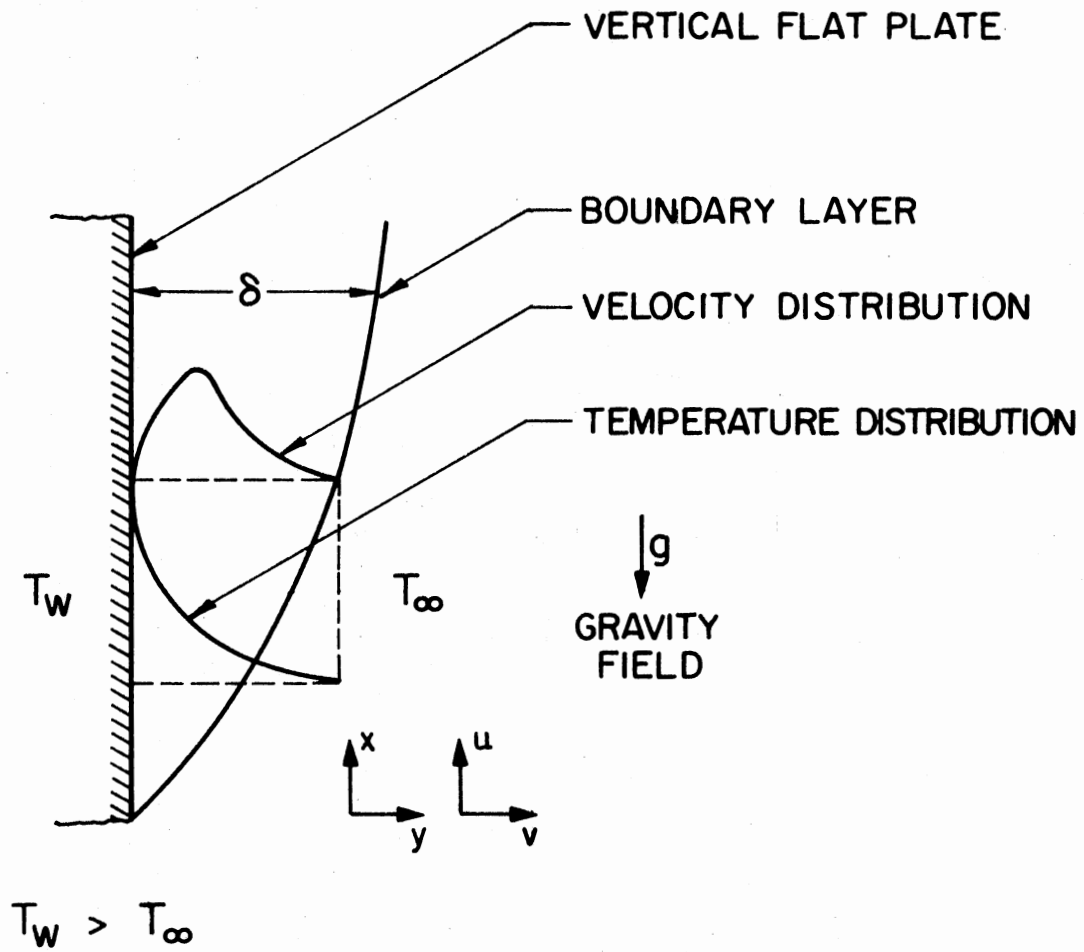


Figure 1. Physical Model and Coordinate System

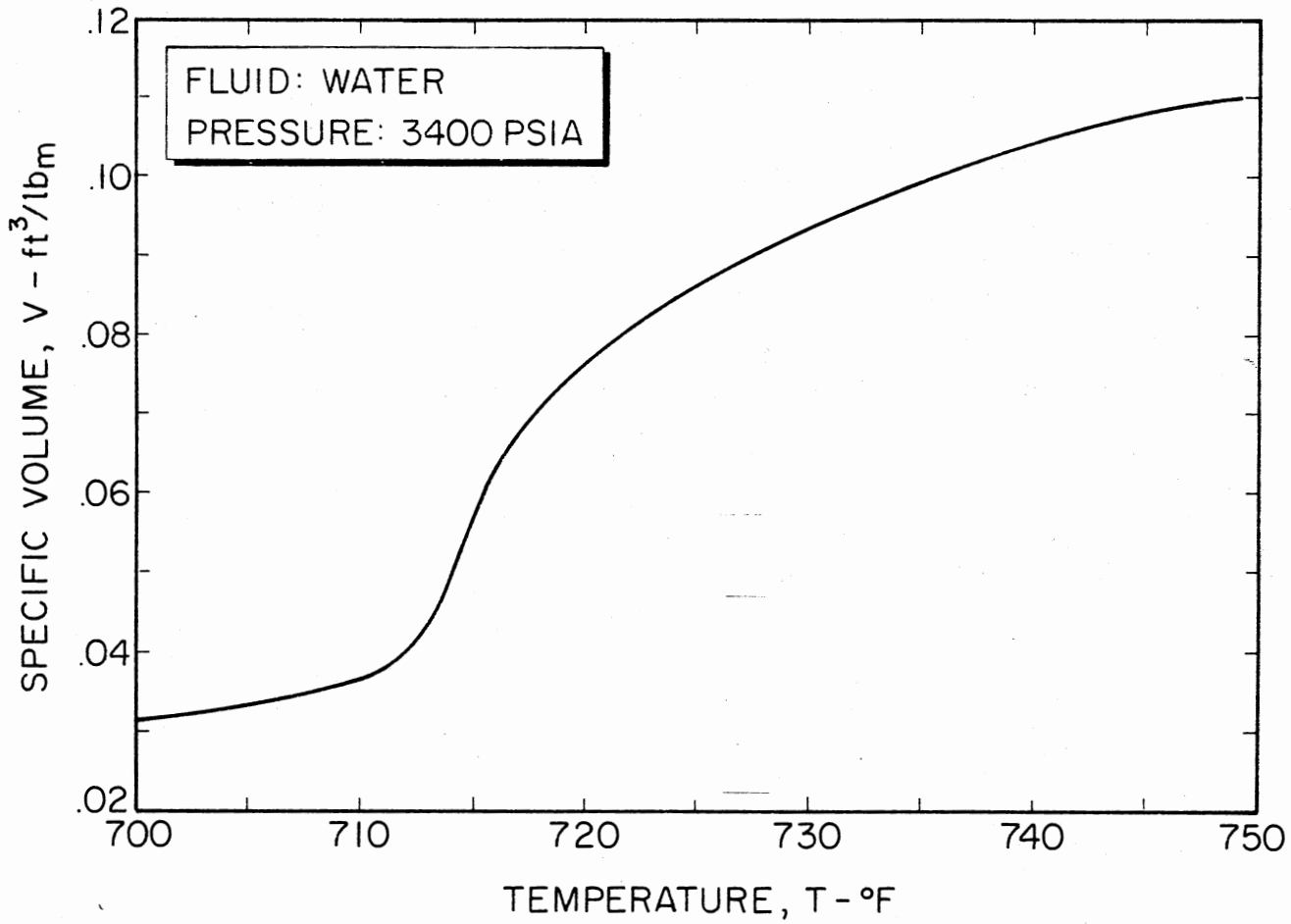


Figure 2. Specific Volume in the Supercritical Region

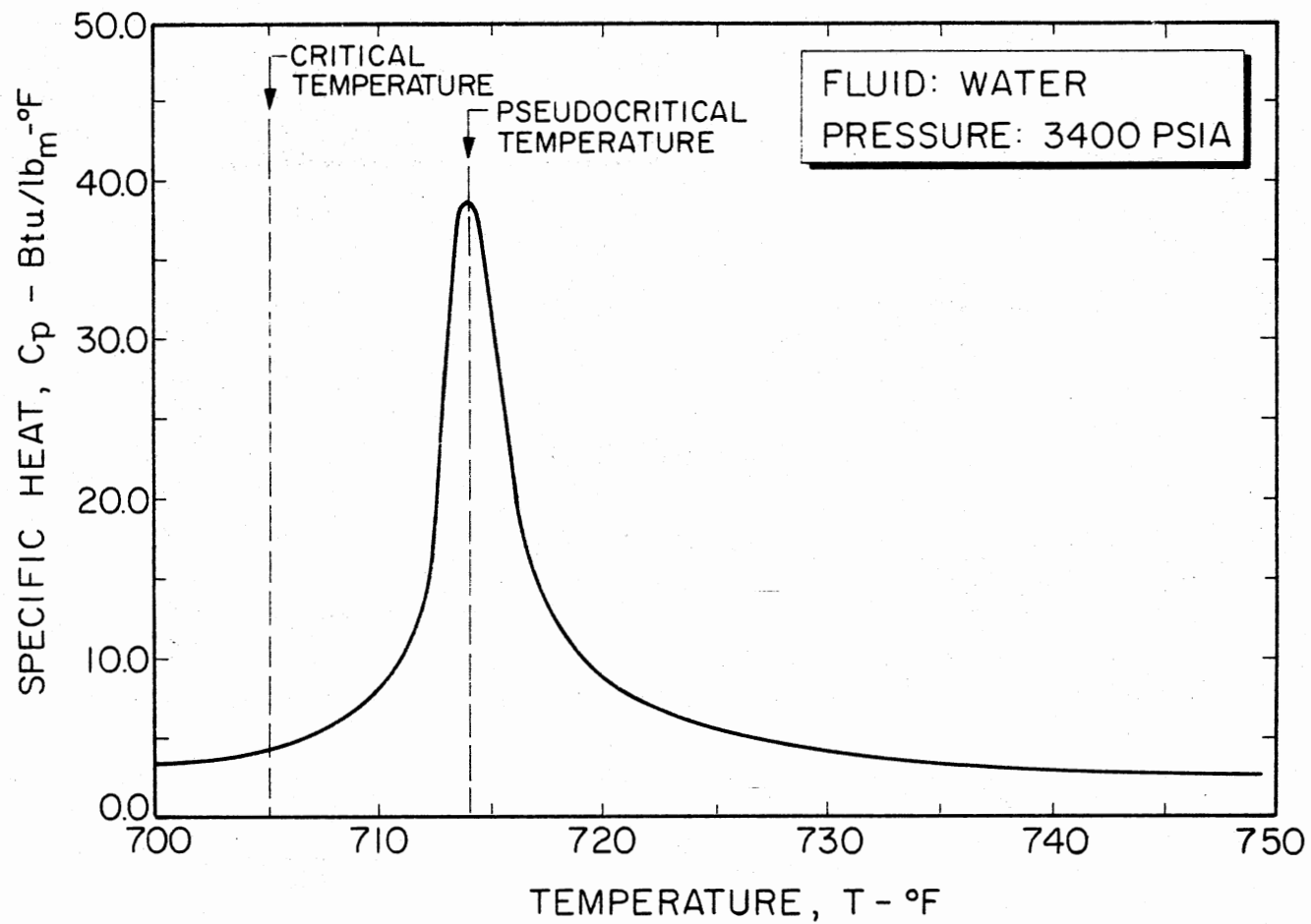


Figure 3. Specific Heat in the Supercritical Region

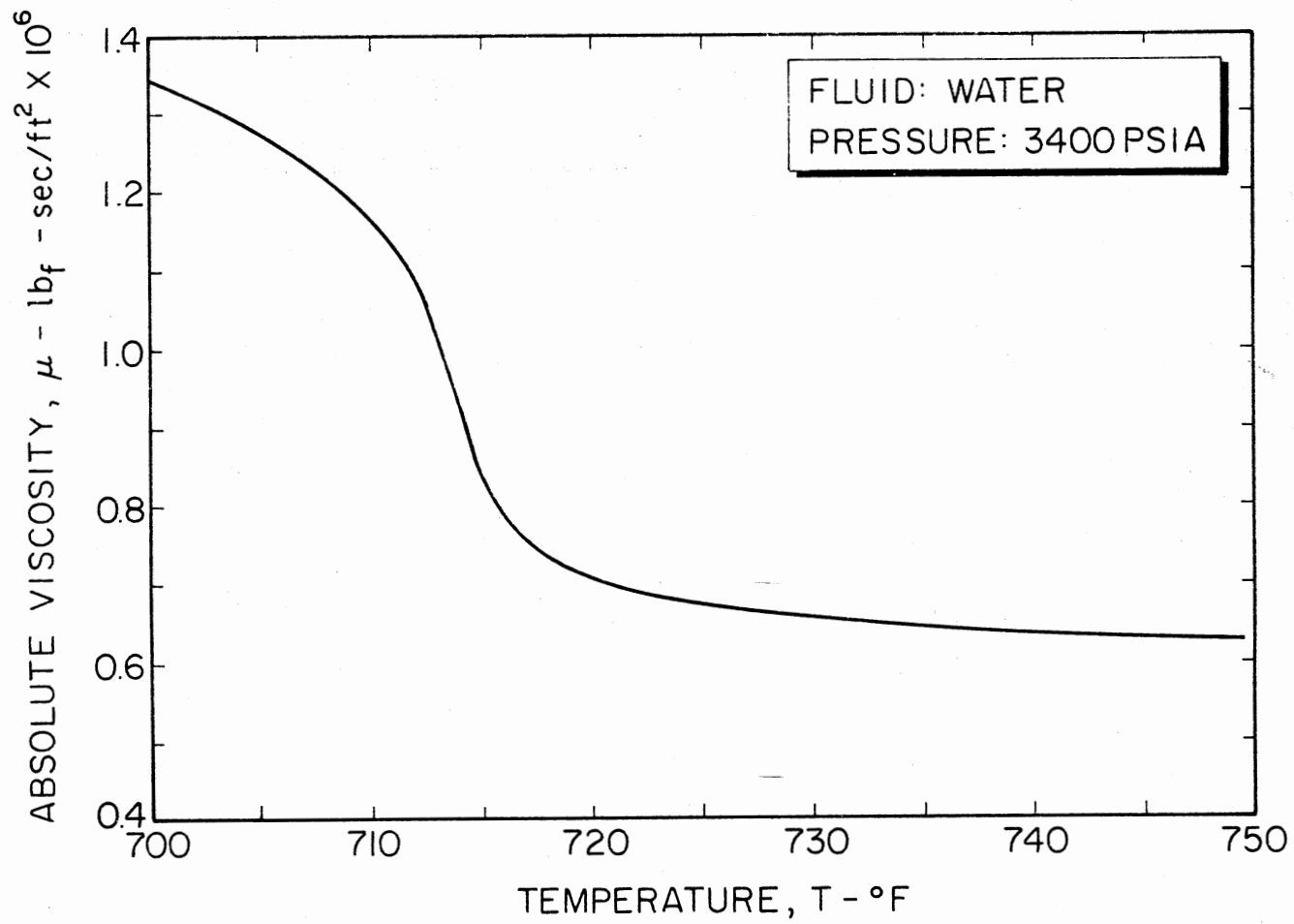


Figure 4. Absolute Viscosity in the Supercritical Region

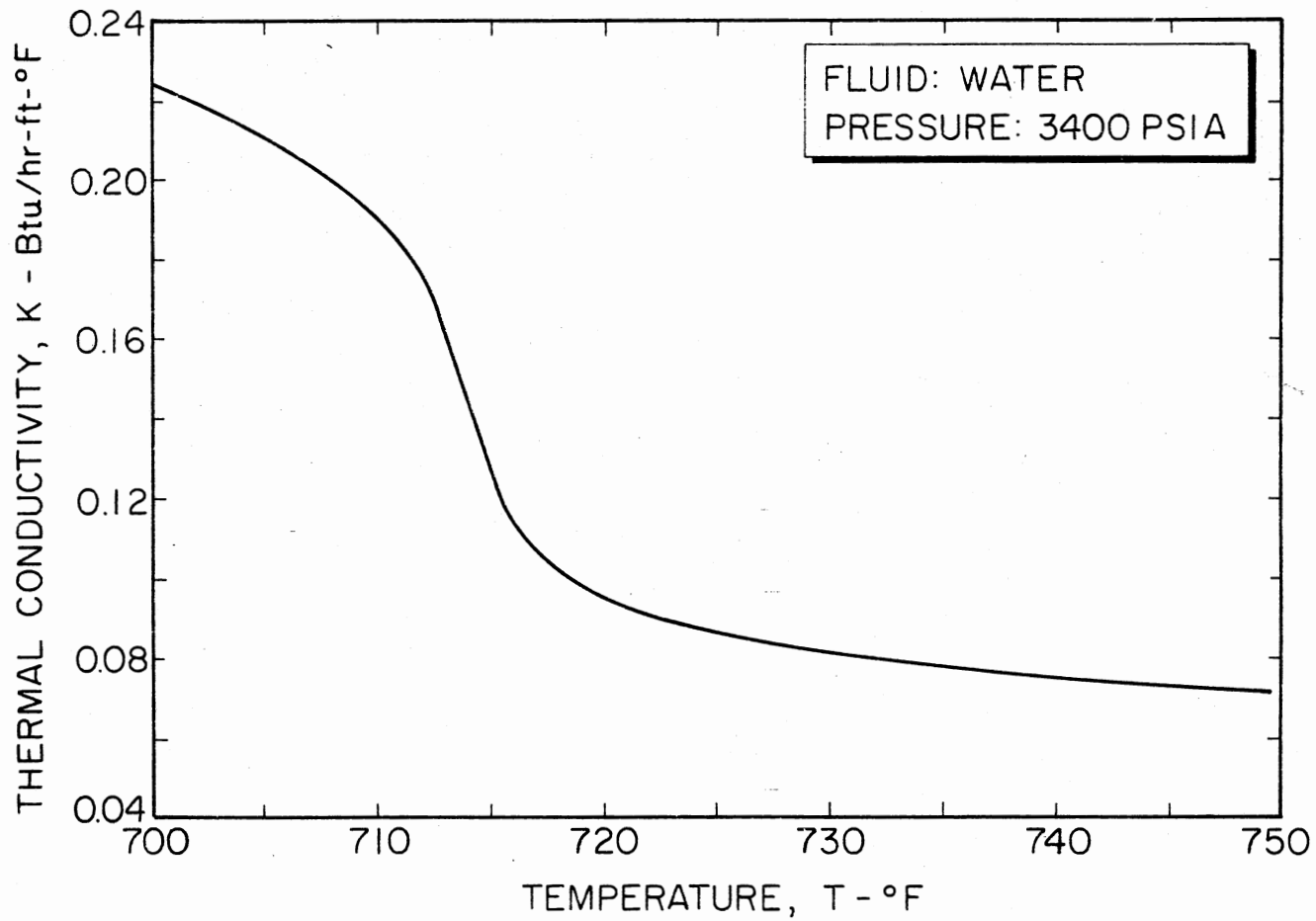


Figure 5. Thermal Conductivity in the Supercritical Region

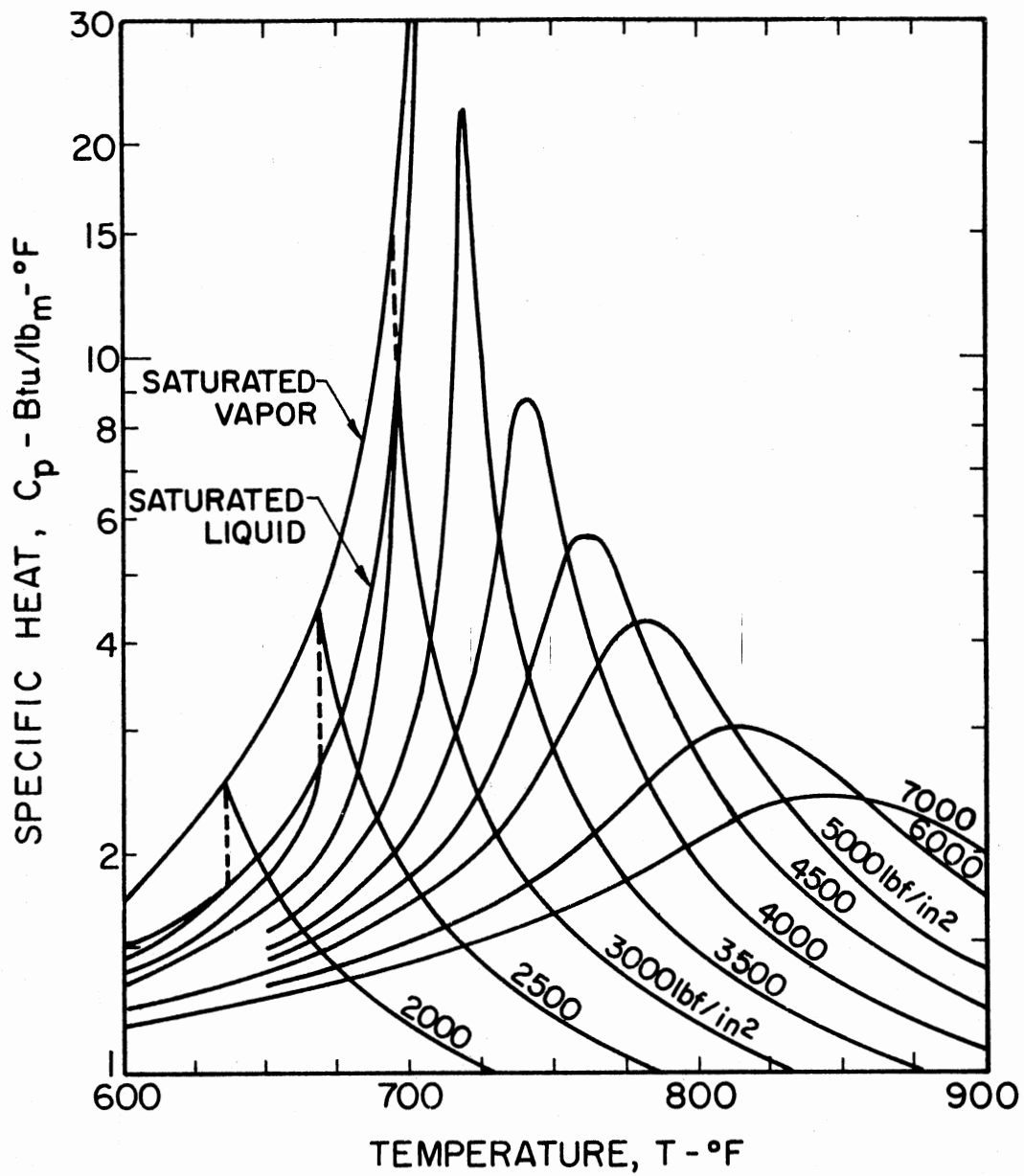


Figure 6. Specific Heat at Constant Pressure Above and Below the Critical Point for Water

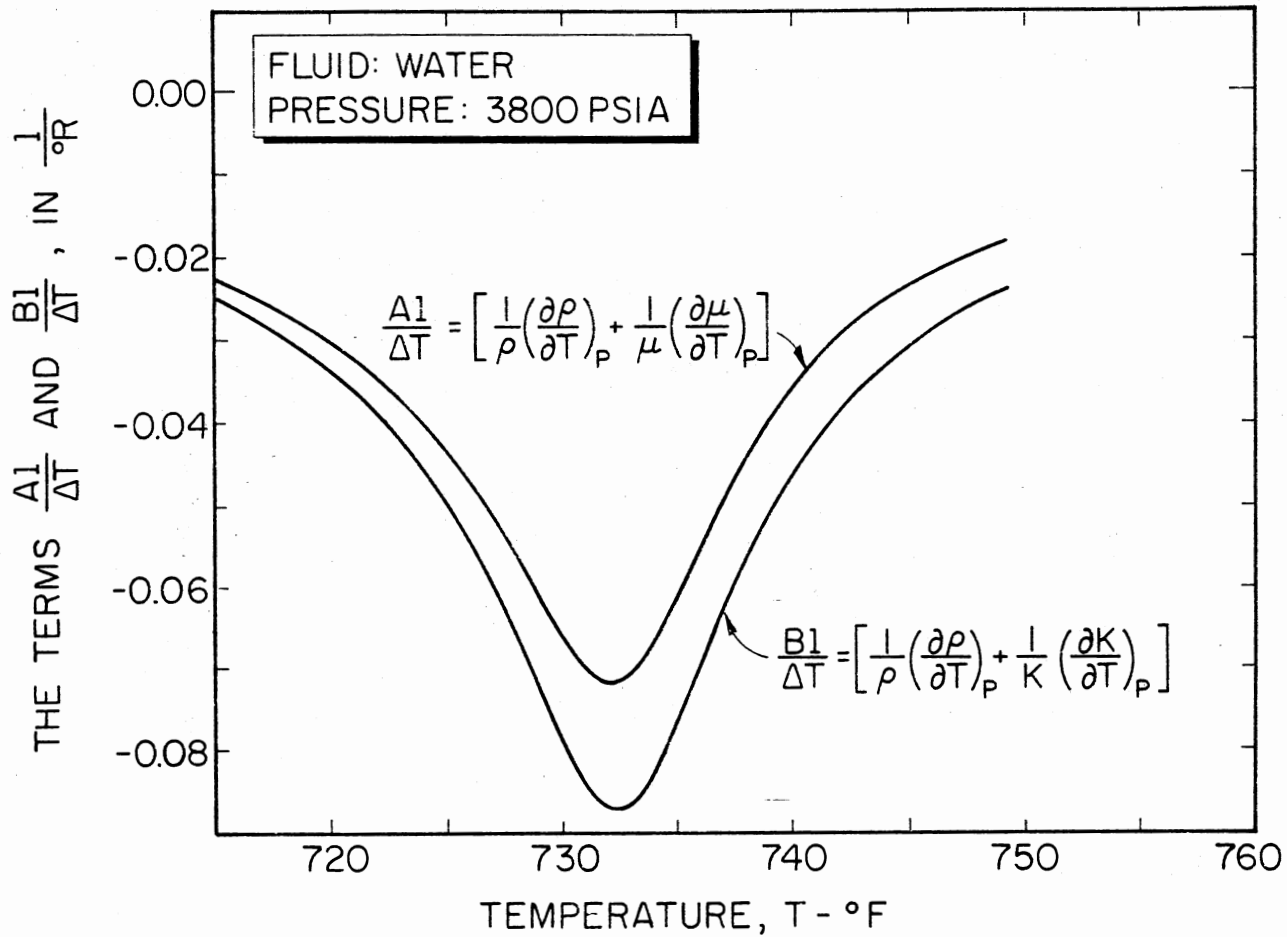


Figure 7. The Terms  $A1/\Delta T$  and  $B1/\Delta T$  (See Equations (3.16) and (3.19))

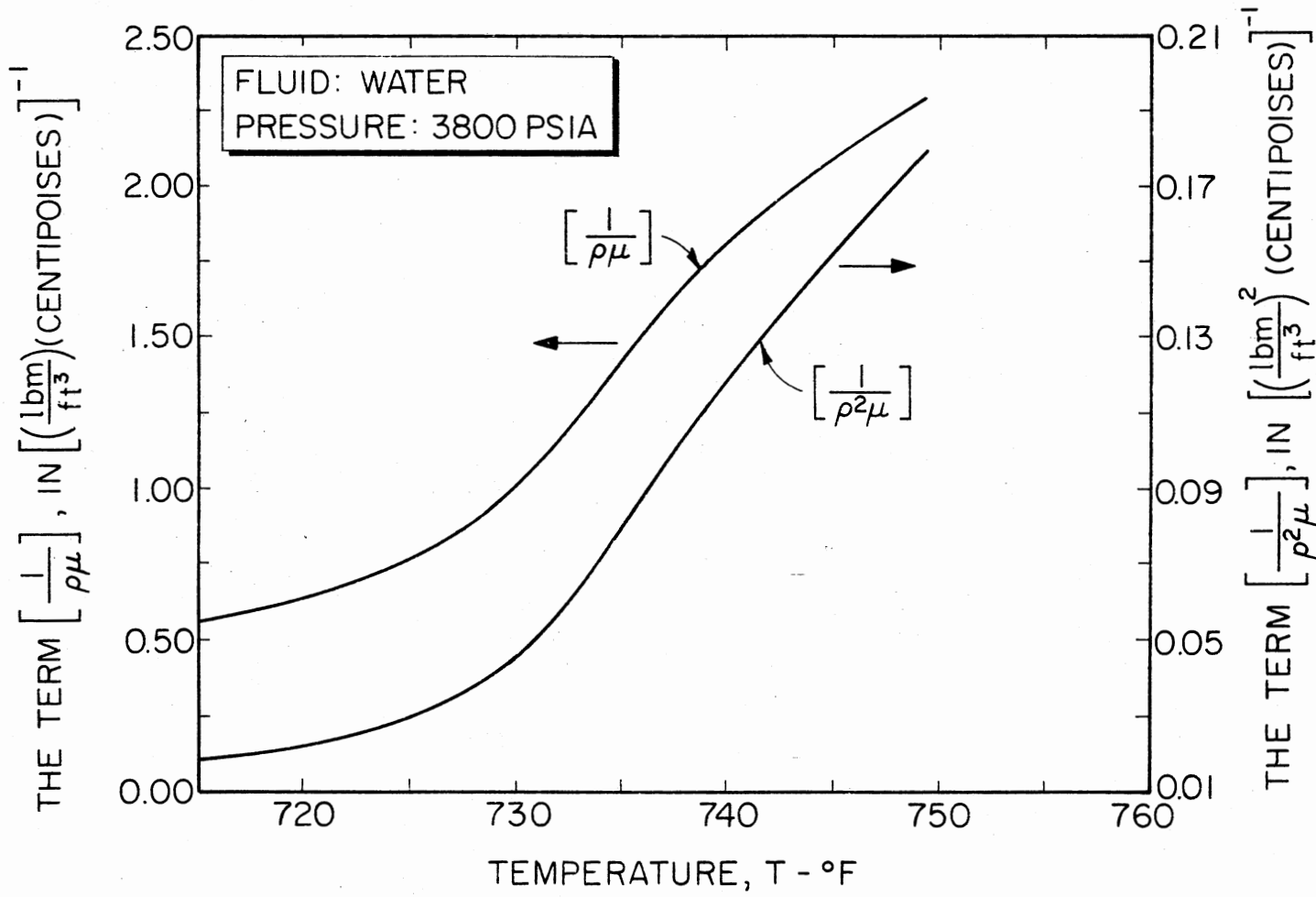


Figure 8. The Terms  $[1/\rho\mu]$  and  $[1/\rho^2\mu]$



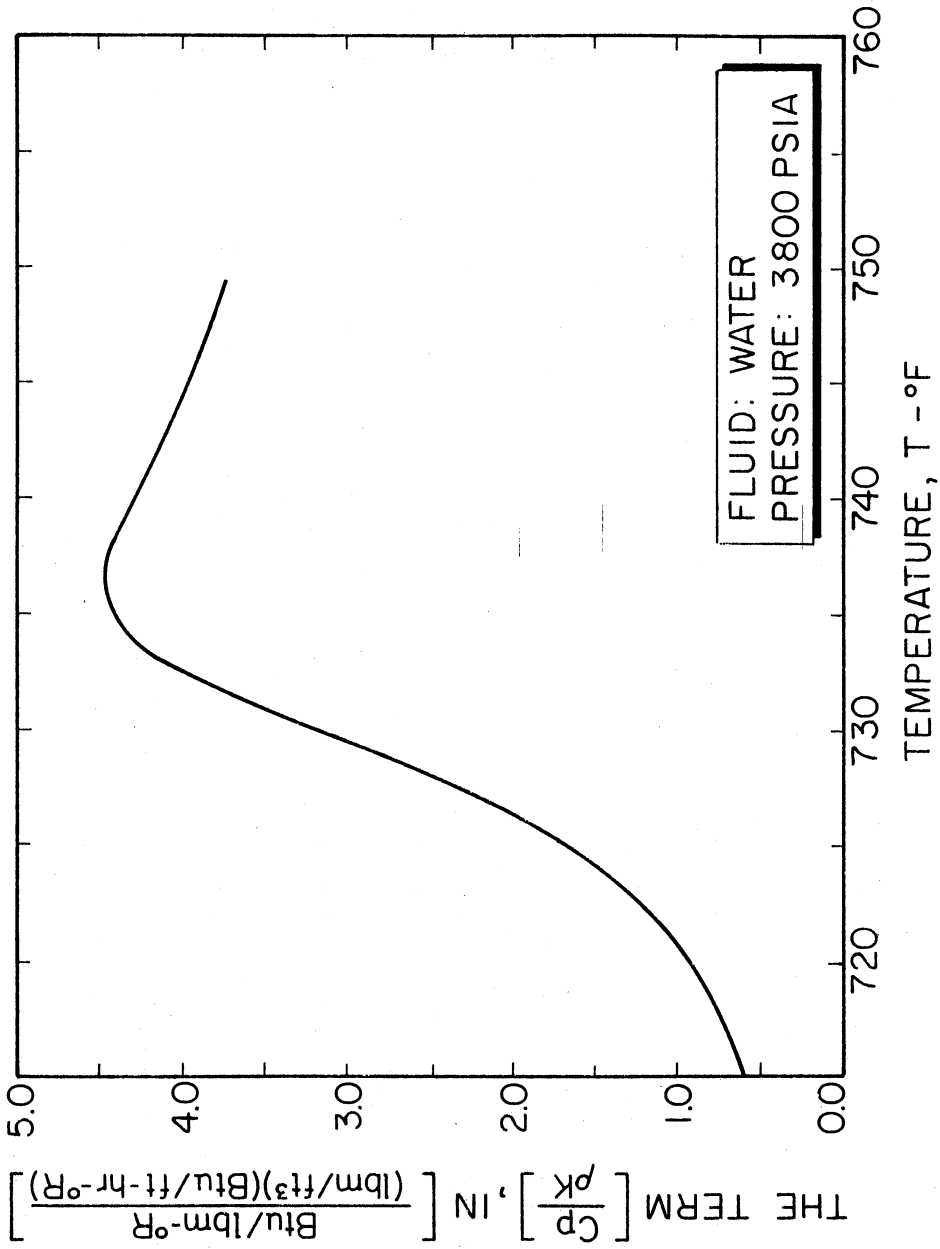


Figure 9. The Term  $[C_p/\rho K]$

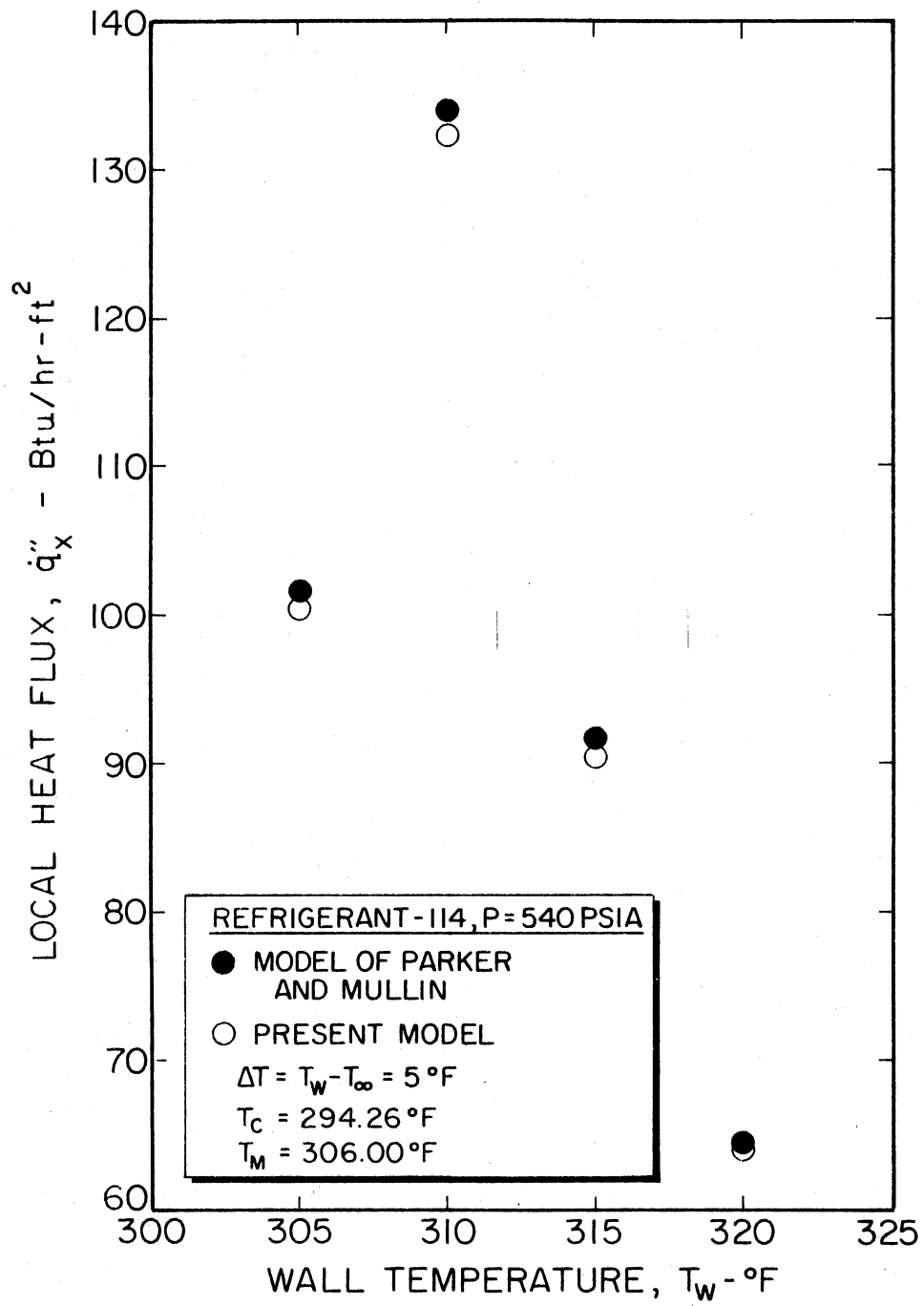


Figure 10. Comparison of Predicted Values of Local Heat Flux With Model of Parker and Mullin (5)

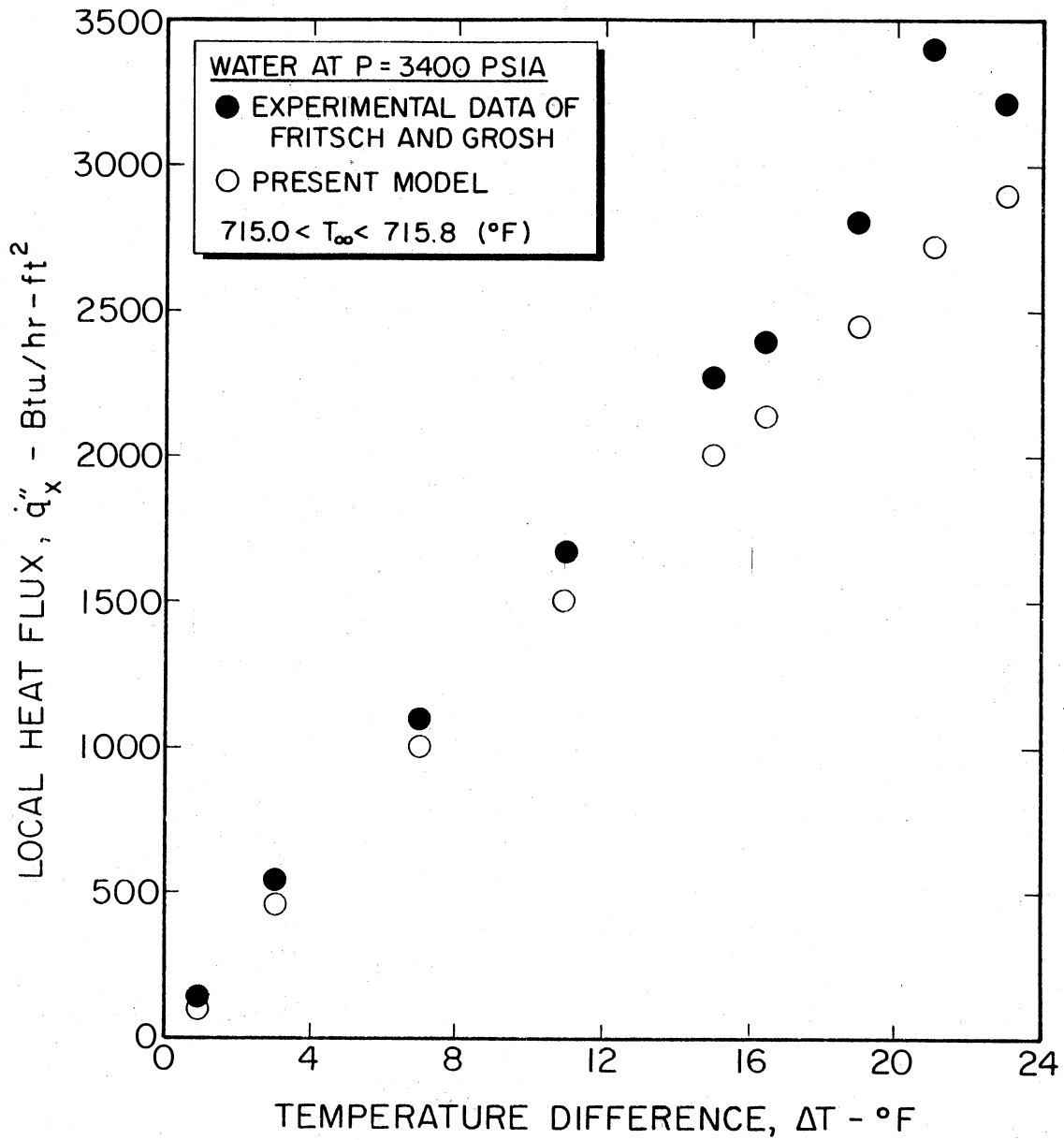


Figure 11. Comparison of Predicted Values of Local Heat Flux With Experimental Data of Fritsch and Grosh (12)

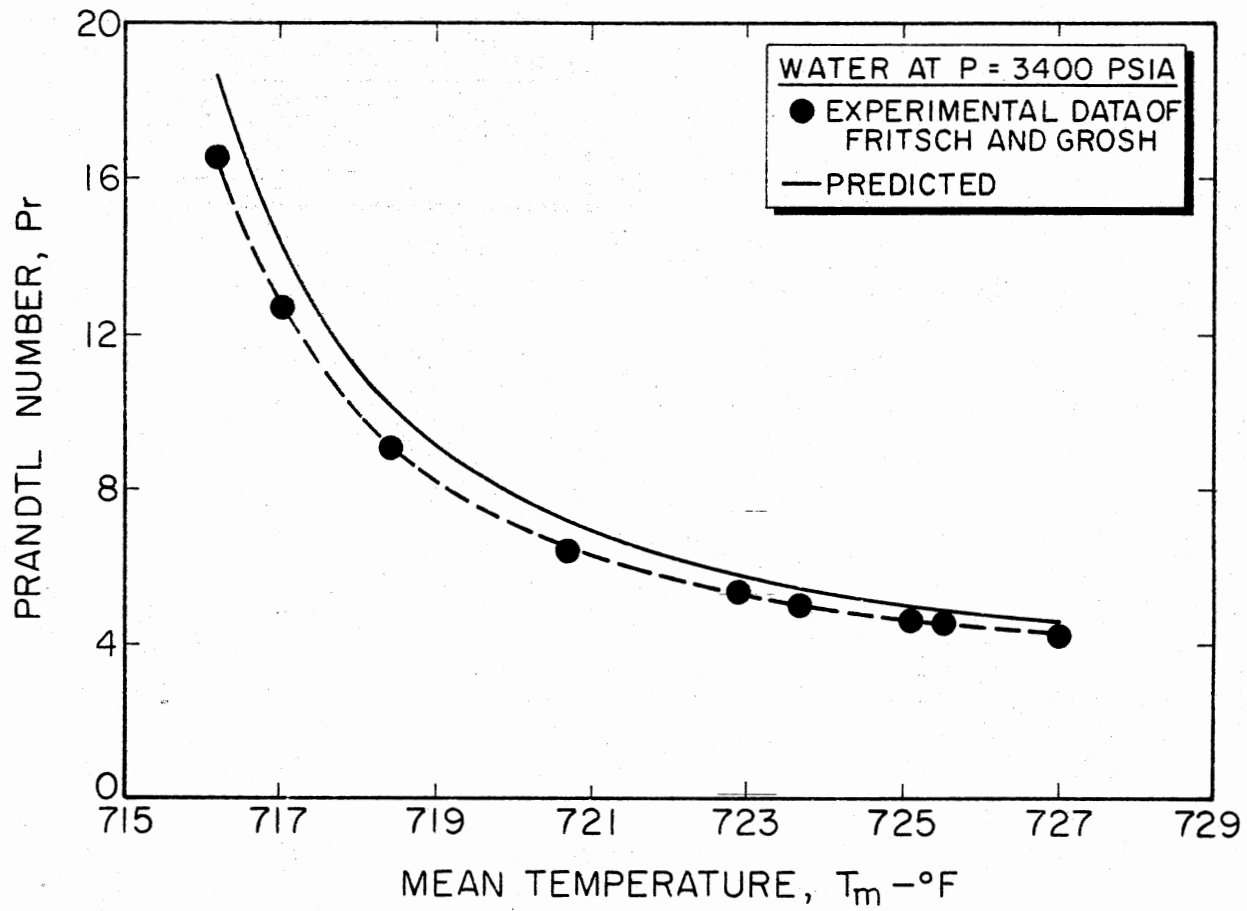


Figure 12. Comparison of Predicted Values of Prandtl Number With Experimental Data of Fritsch and Grosh (12)

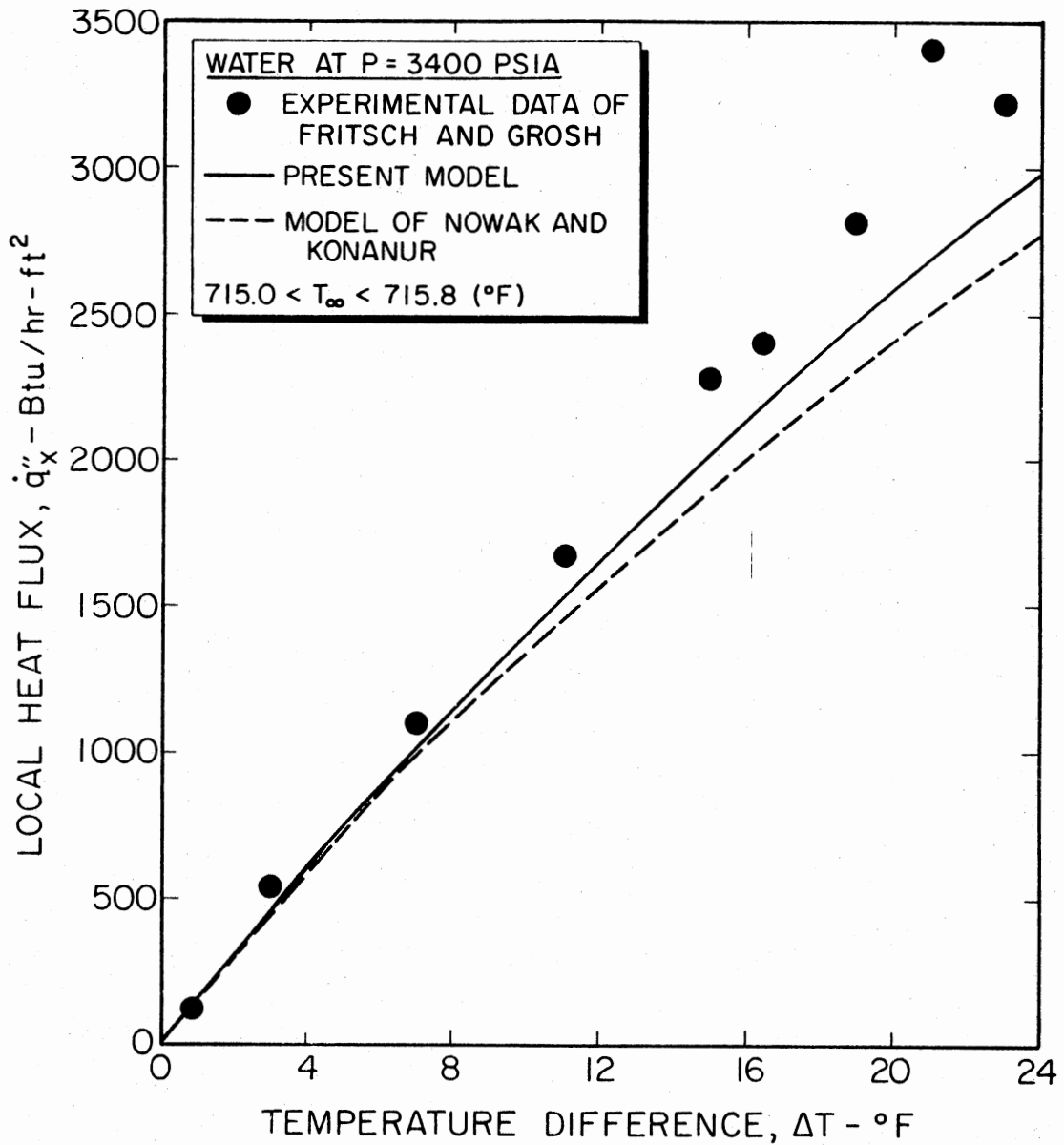


Figure 13. Comparison of Present Model With Model of Nowak and Konanur (7) (Experimental Data From Fritsch and Grosh (12))

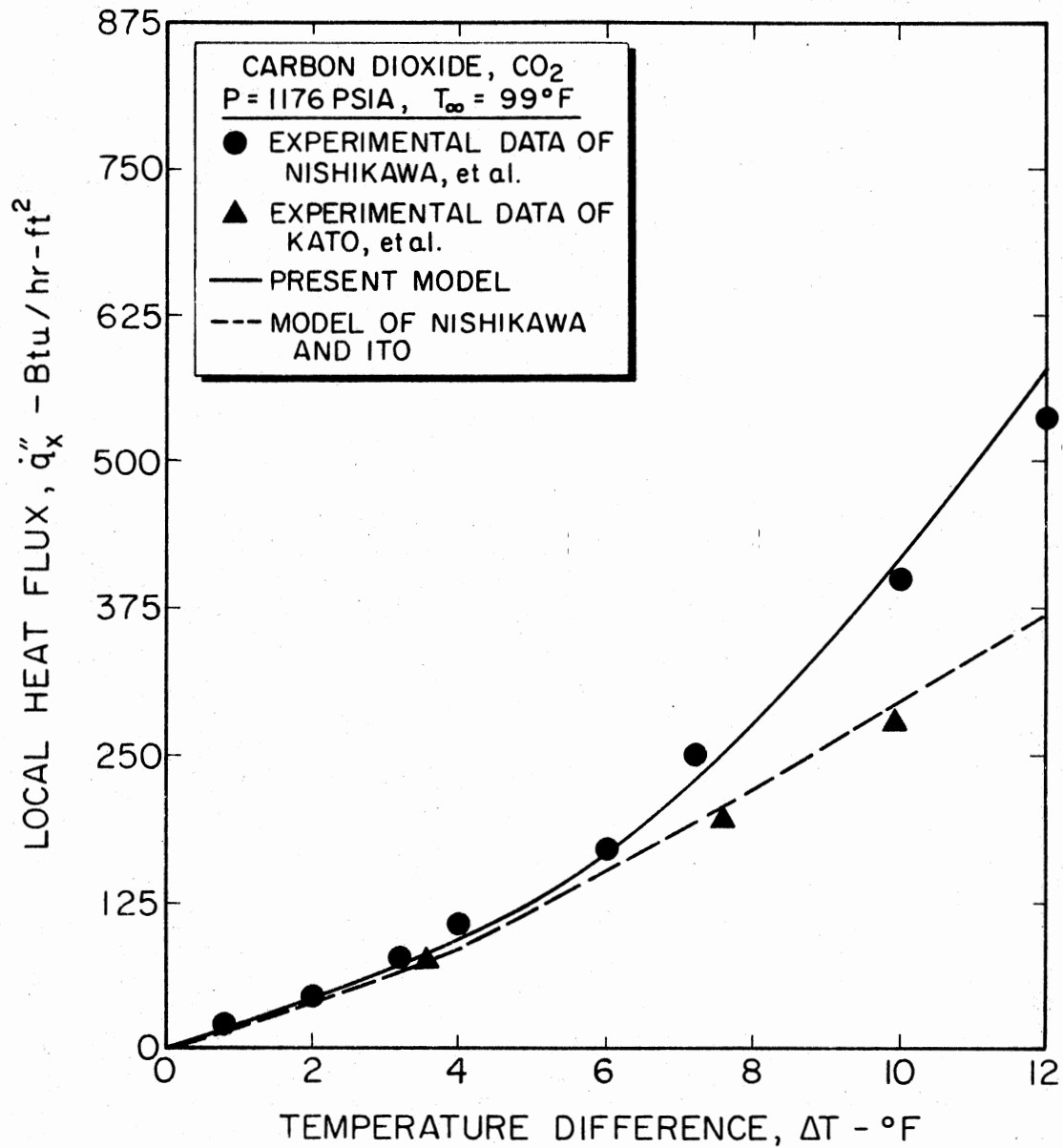


Figure 14. Comparison of Present Model With Experimental Data of Nishikawa et al. (30) and Kato et al. (29), and Model of Nishikawa and Ito (6)

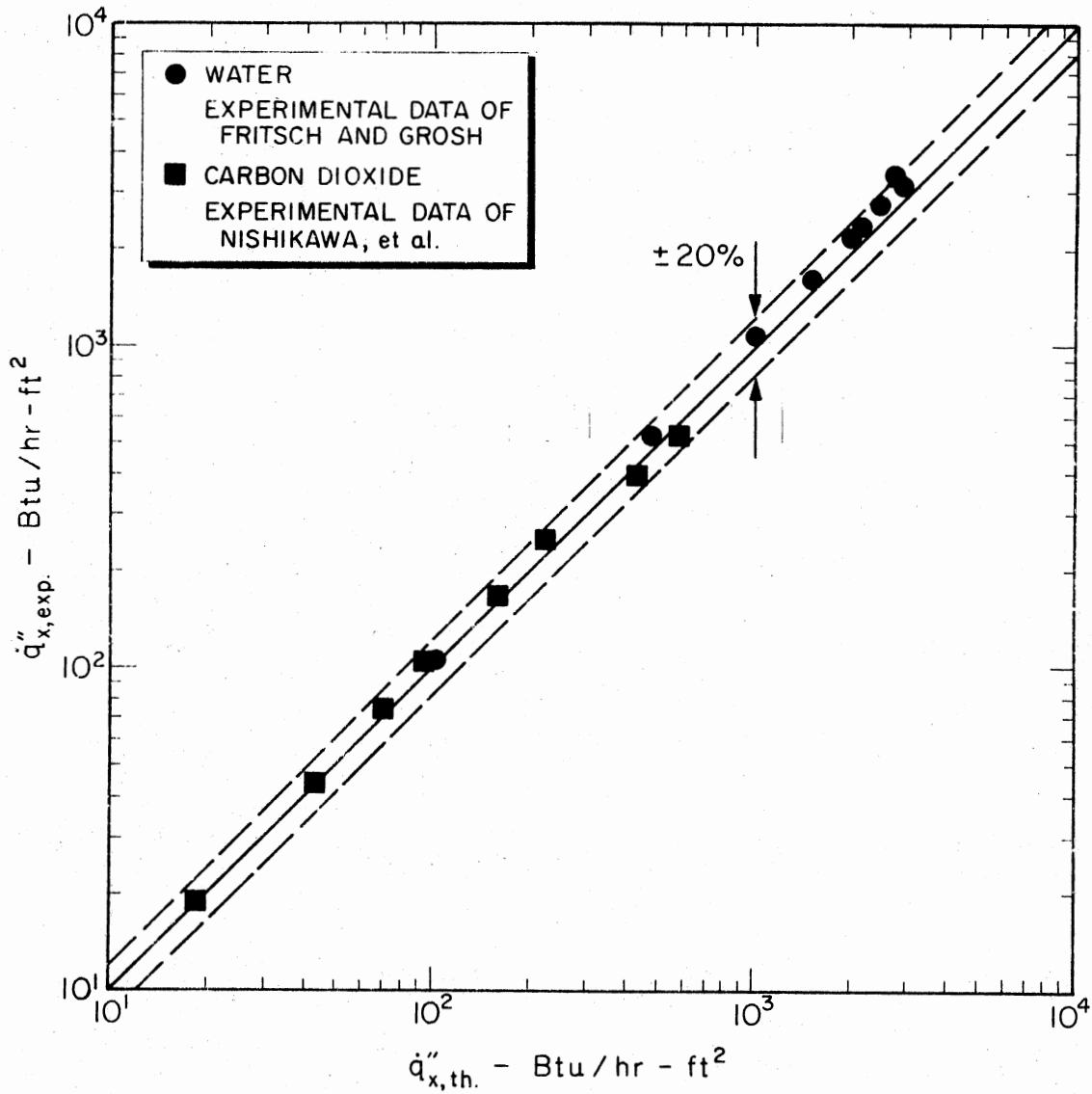


Figure 15. Comparison of Predicted Local Heat Flux Using the Present Model With Measurements

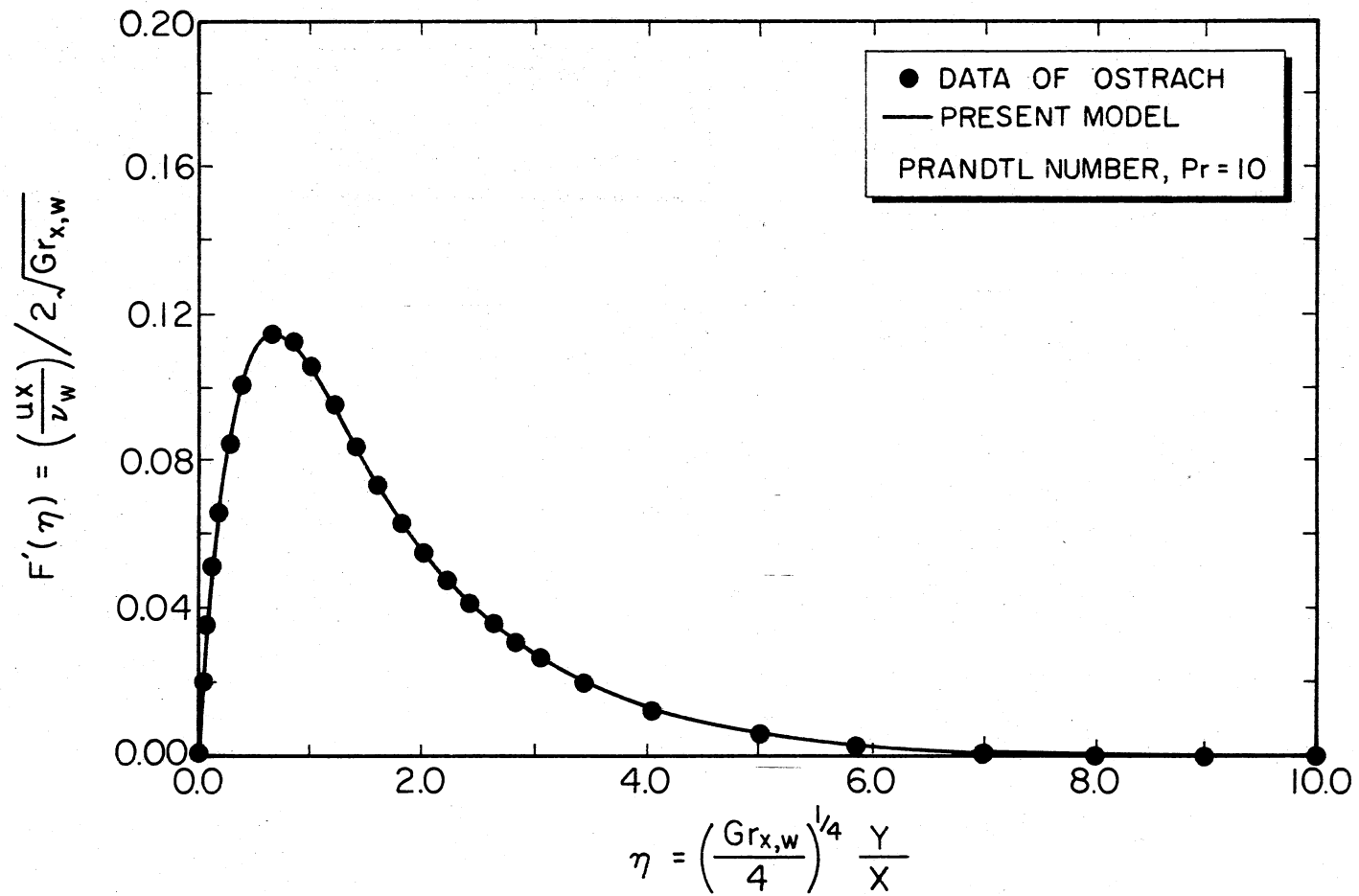


Figure 16. Comparison of Predicted Velocity Profile With Data of Ostrach (10), for the Constant Property Case



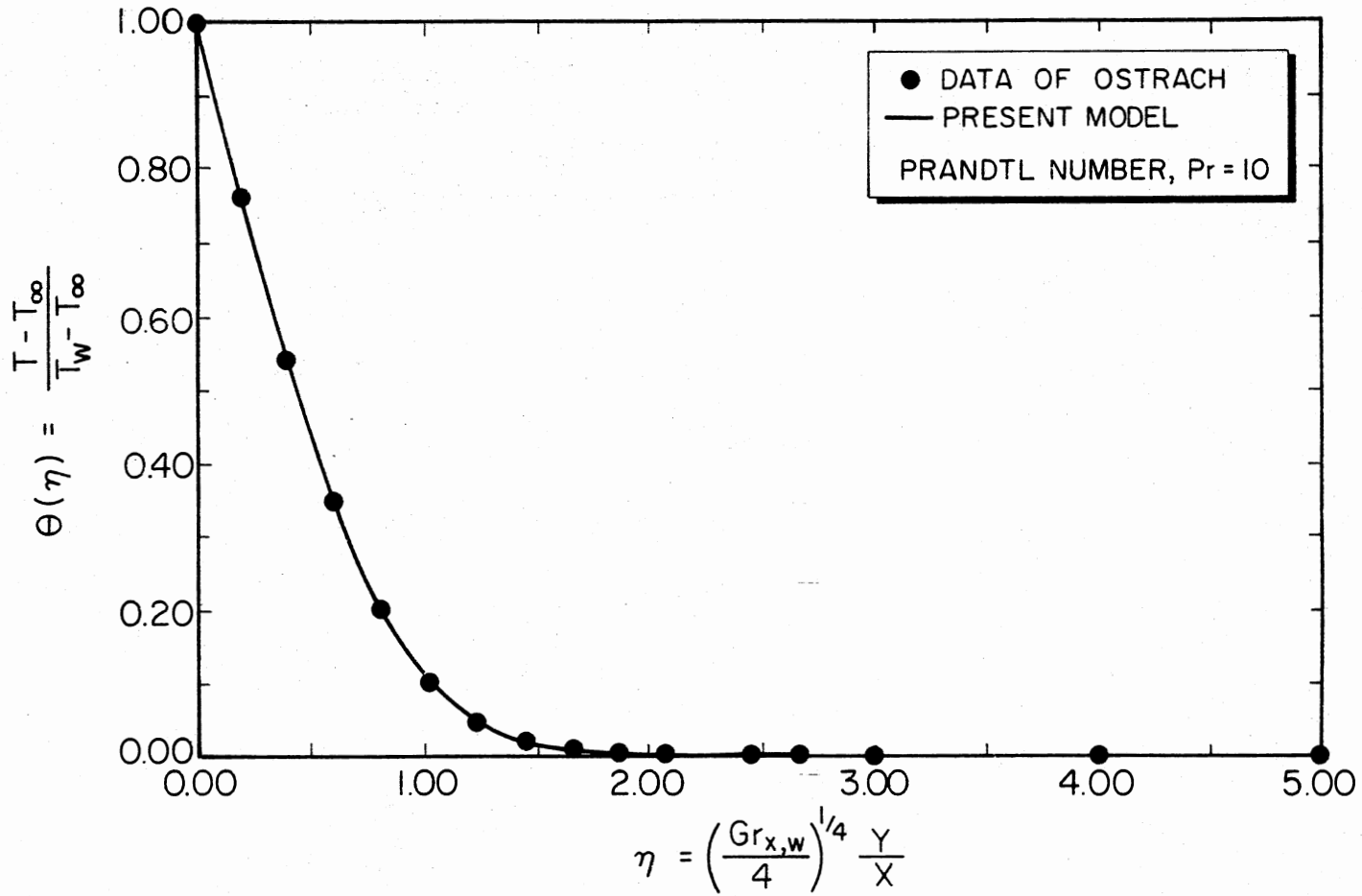


Figure 17. Comparison of Predicted Dimensionless Temperature Profile With Data of Ostrach (10), for the Constant Property Case

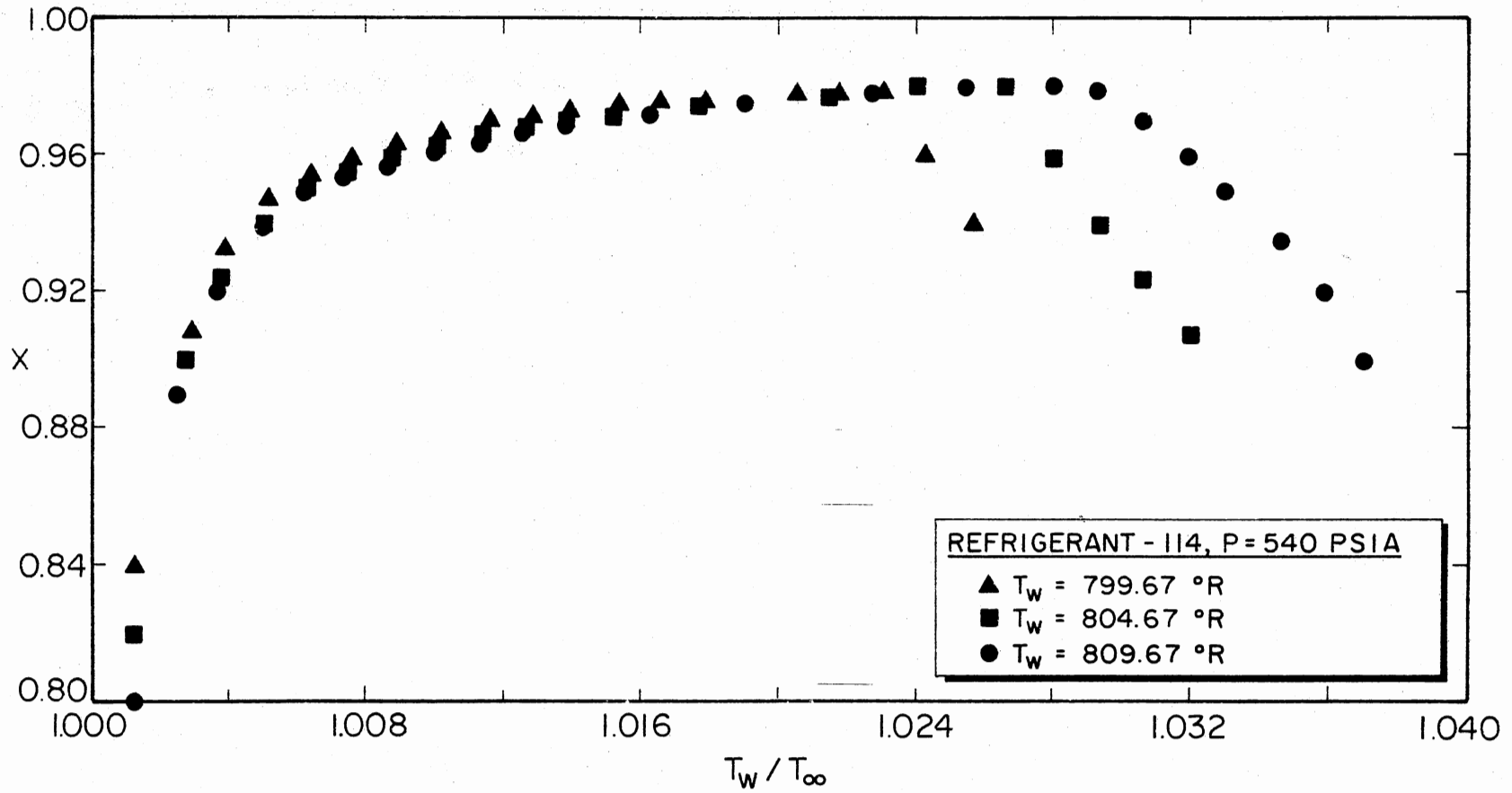


Figure 18. Variation of  $x(T_x = T_w - x(T_w - T_\infty))$  With Wall Temperature and  $T_w/T_\infty$

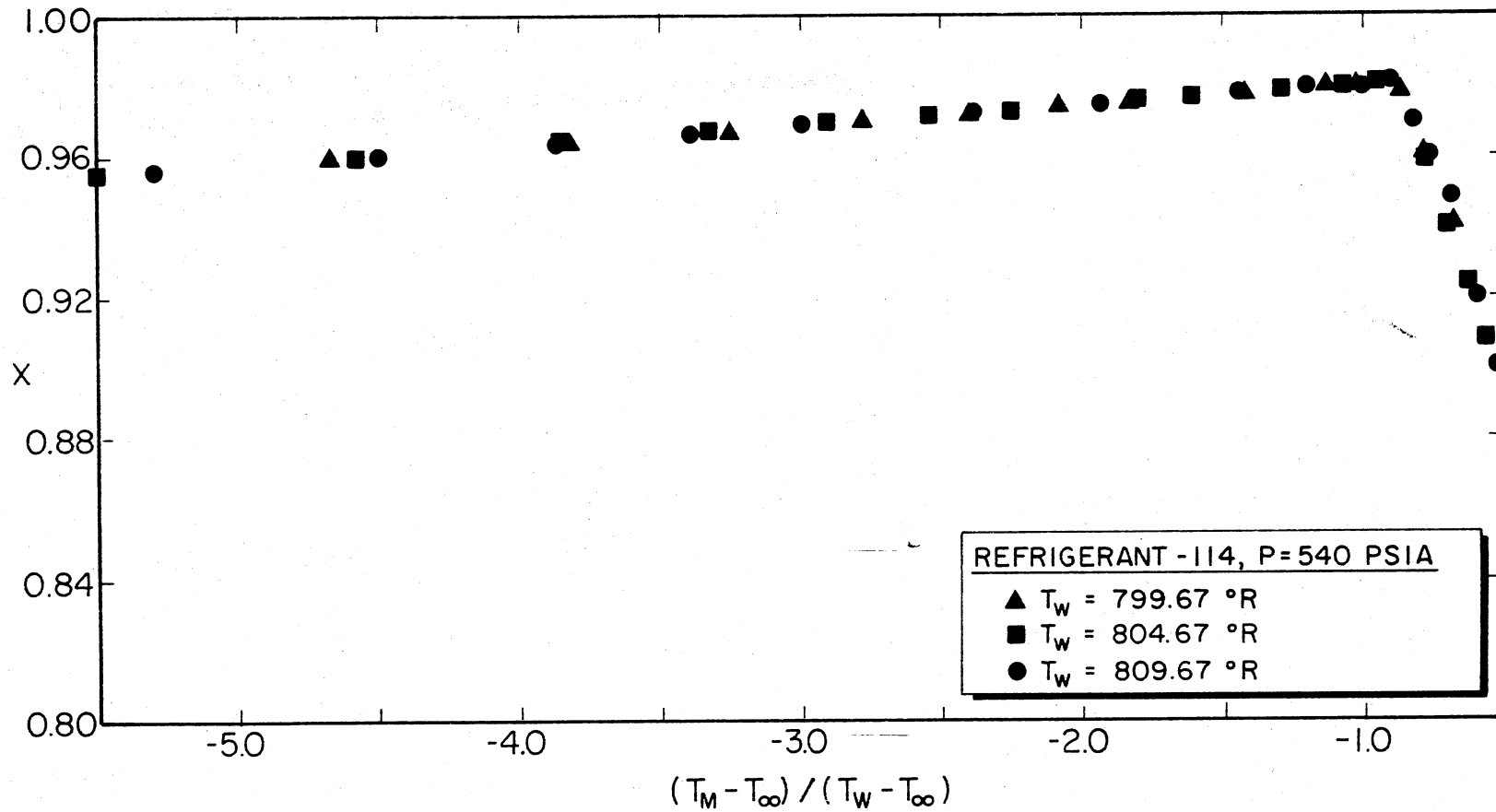


Figure 19. Variation of  $x(T_x = T_w - x(T_w - T_\infty))$  With  $(T_M - T_\infty)/(T_w - T_\infty)$  at Various Temperatures and a Single Pressure

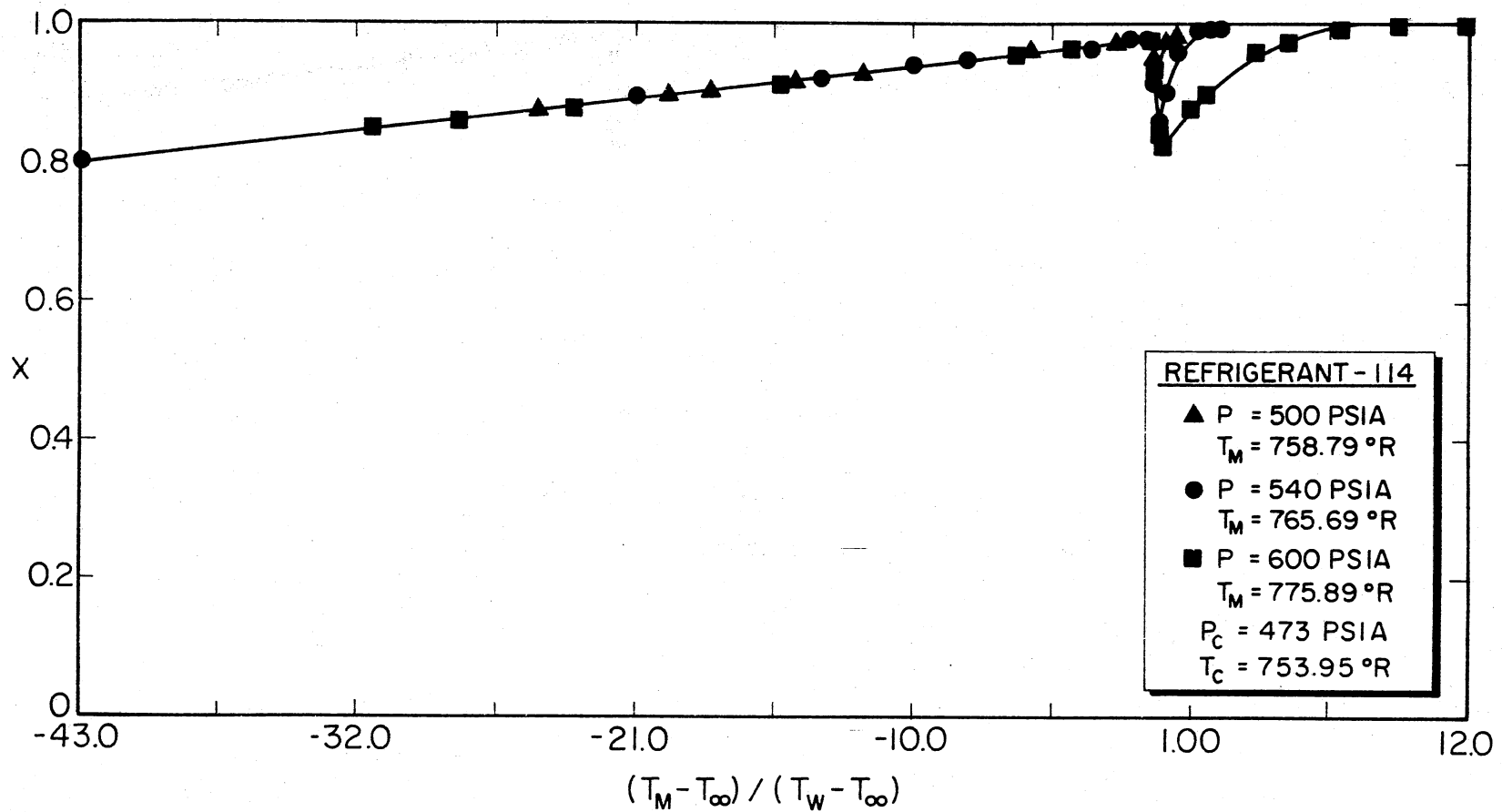


Figure 20. Variation of  $x(T_x = T_w - x(T_w - T_\infty))$  With  $(T_M - T_\infty)/(T_W - T_\infty)$  at Various Temperatures and Pressures

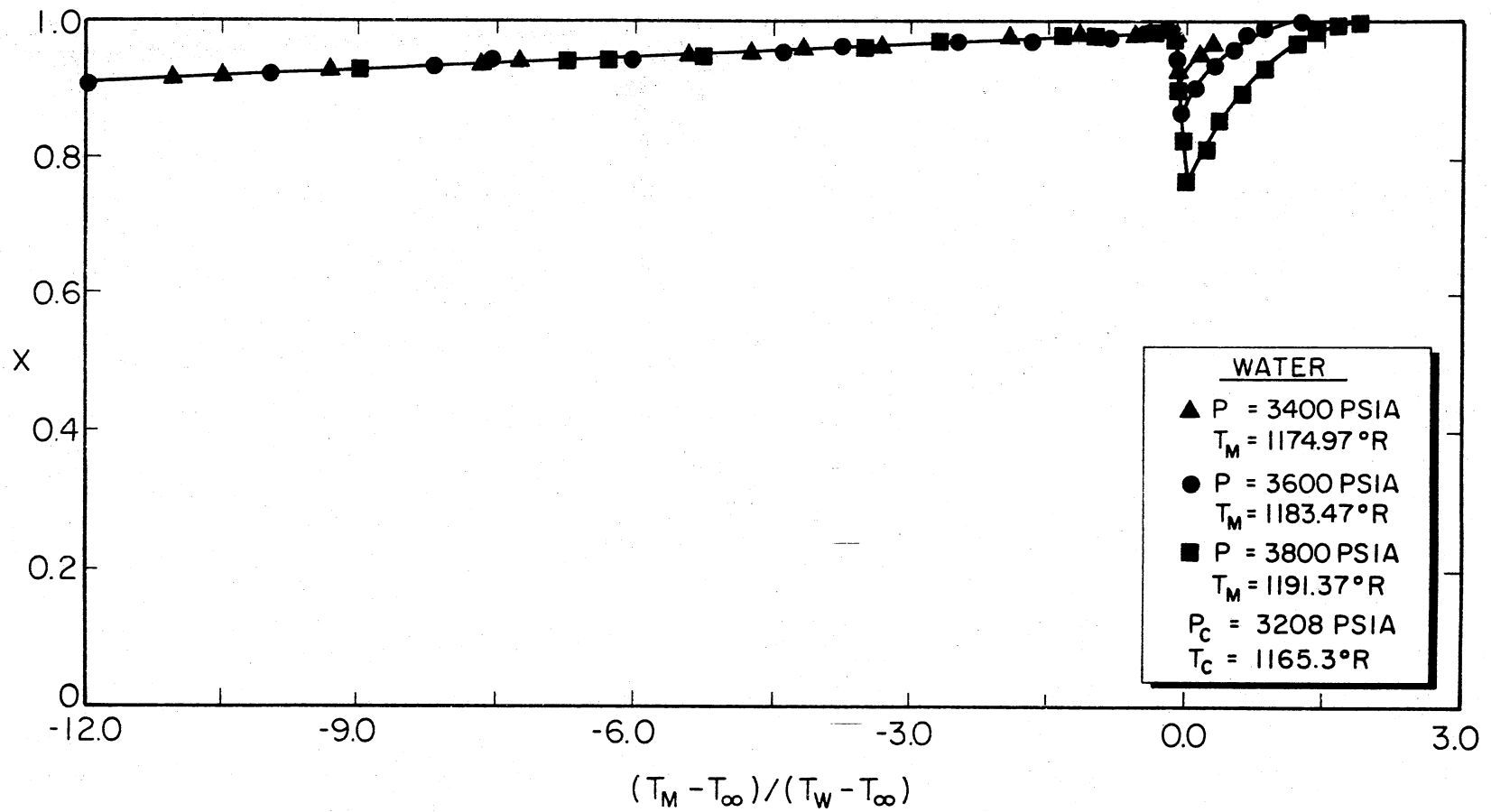


Figure 21. Variation of  $x(T_x = T_w - x(T_w - T_\infty))$  With  $(T_M - T_\infty)/(T_w - T_\infty)$  at Various Temperatures and Pressures

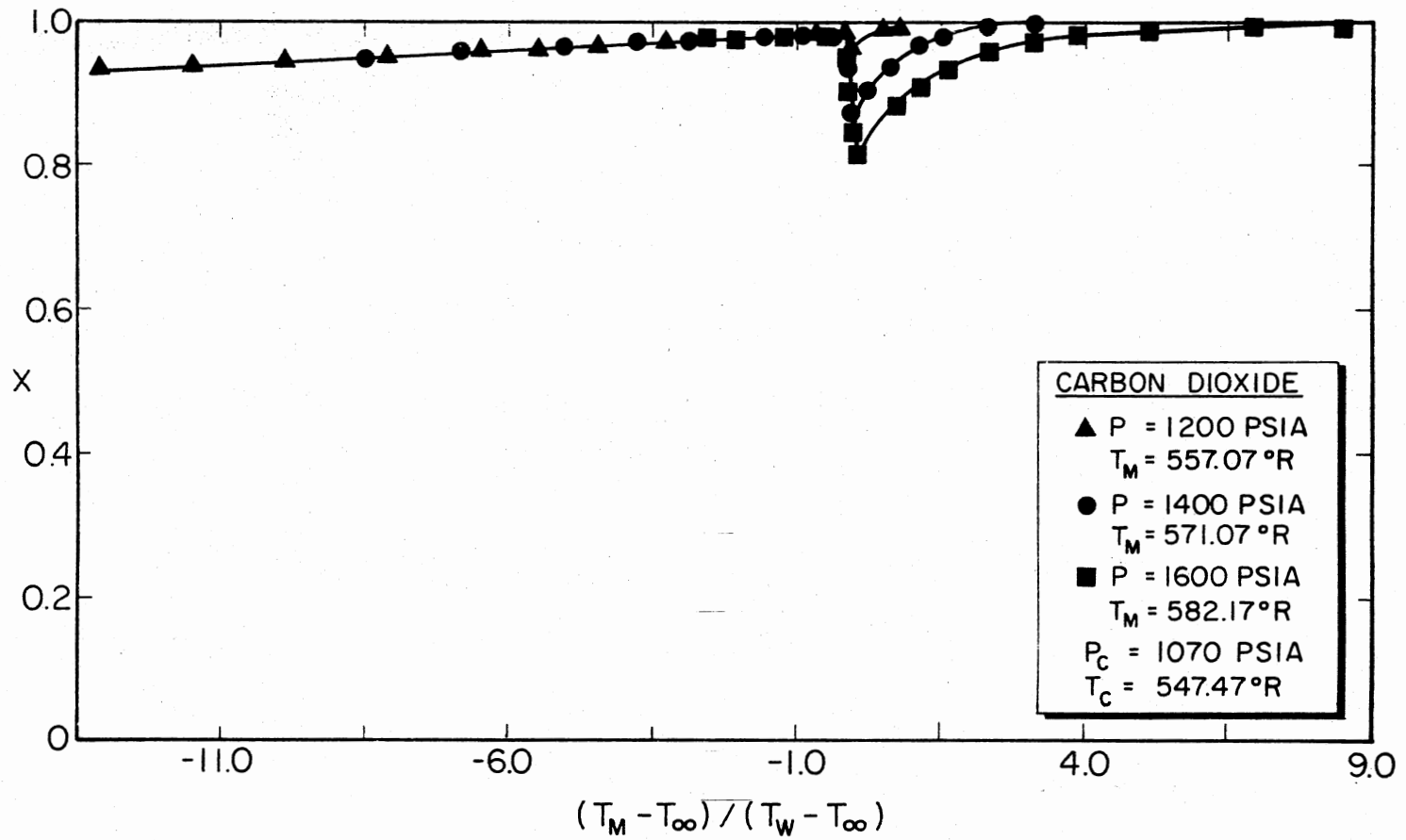


Figure 22. Variation of  $x(T_x = T_w - x(T_w - T_\infty))$  With  $(T_M - T_\infty)/(T_W - T_\infty)$  at Various Temperatures and Pressures

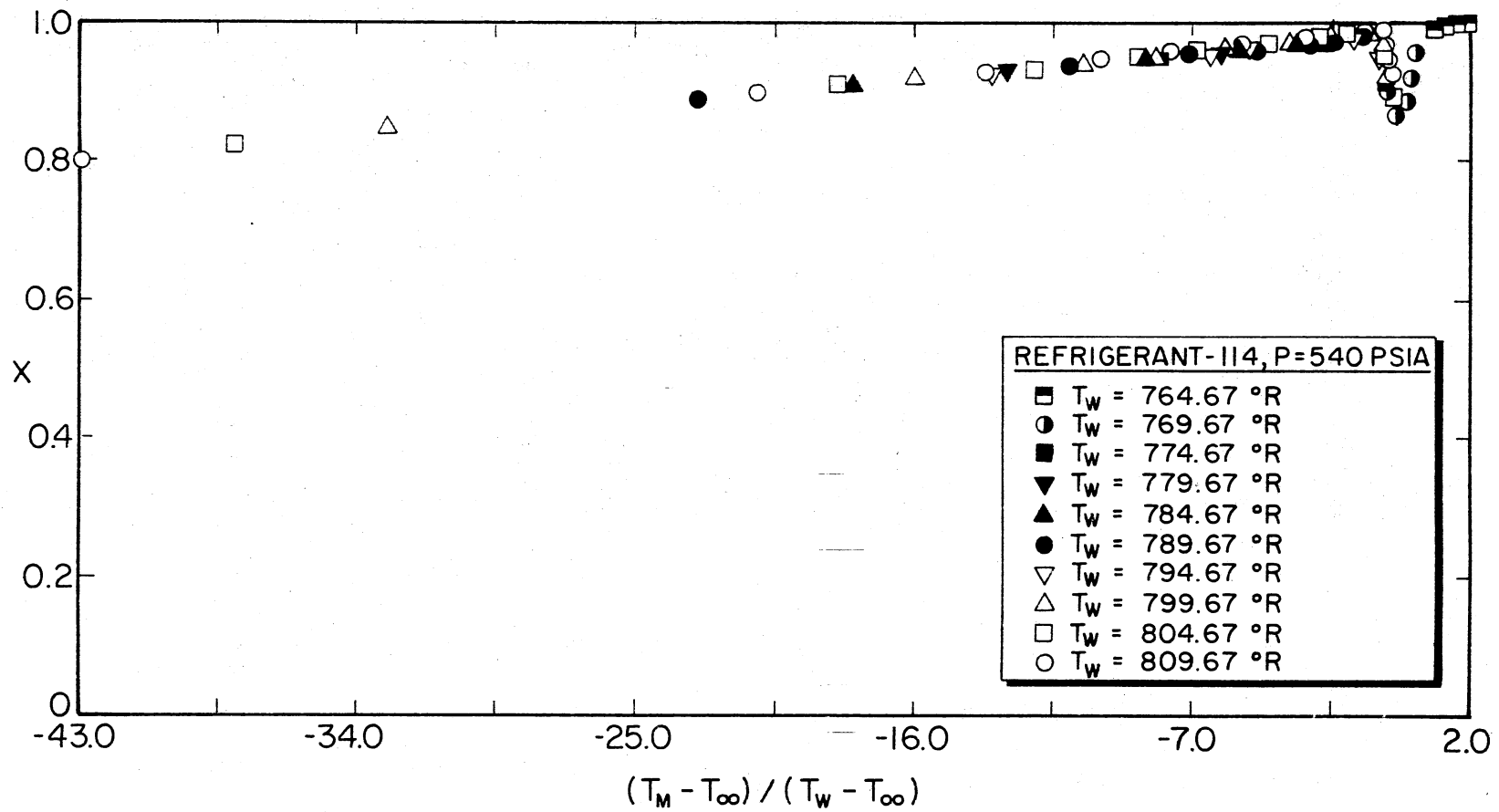


Figure 23. Variation of  $x(T_x = T_W - x(T_W - T_\infty))$  With  $(T_M - T_\infty)/(T_W - T_\infty)$  at Various Temperatures and a Single Pressure

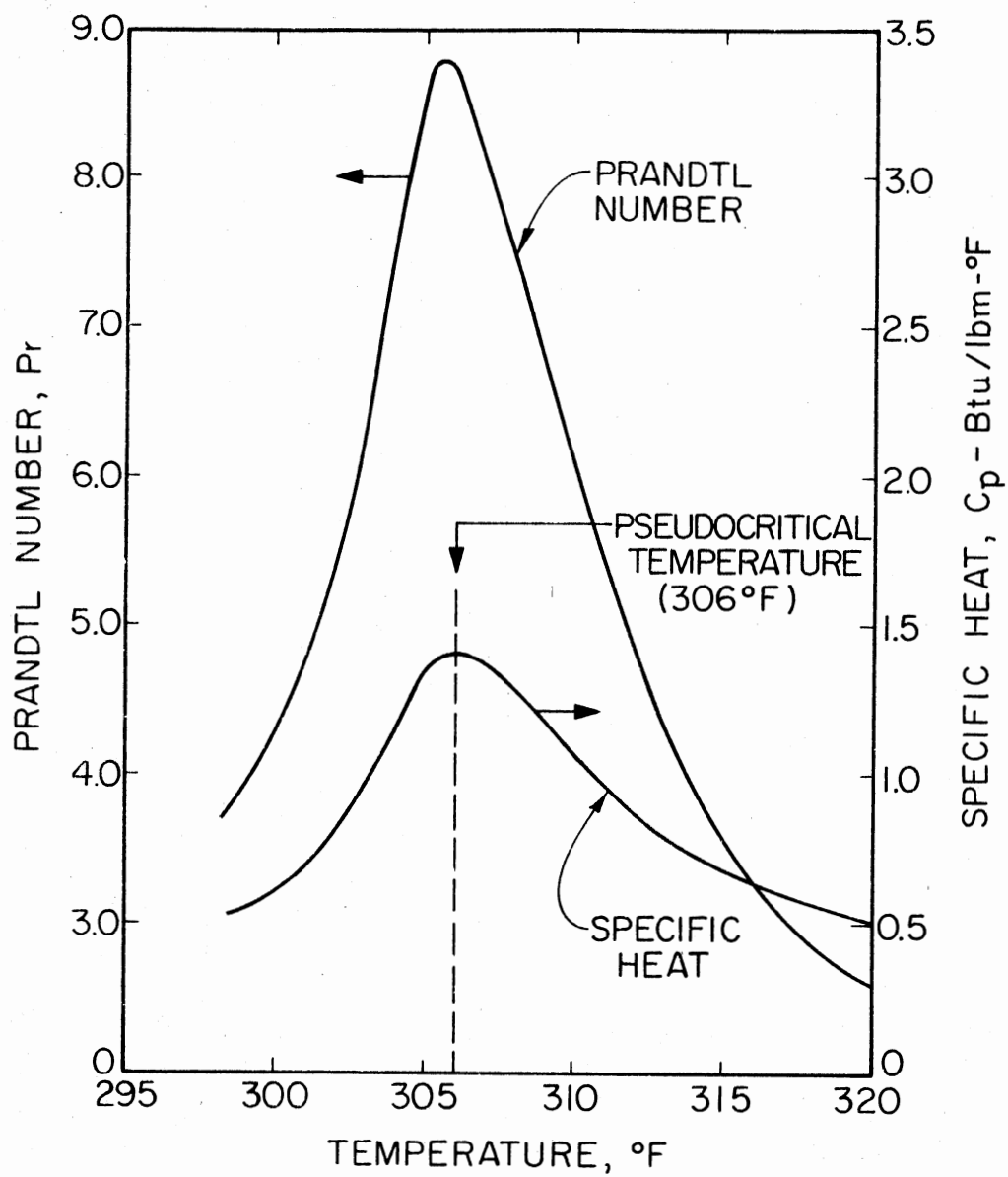


Figure 24. Thermodynamic Properties of Refrigerant-114 in the Supercritical Region at 540 psia



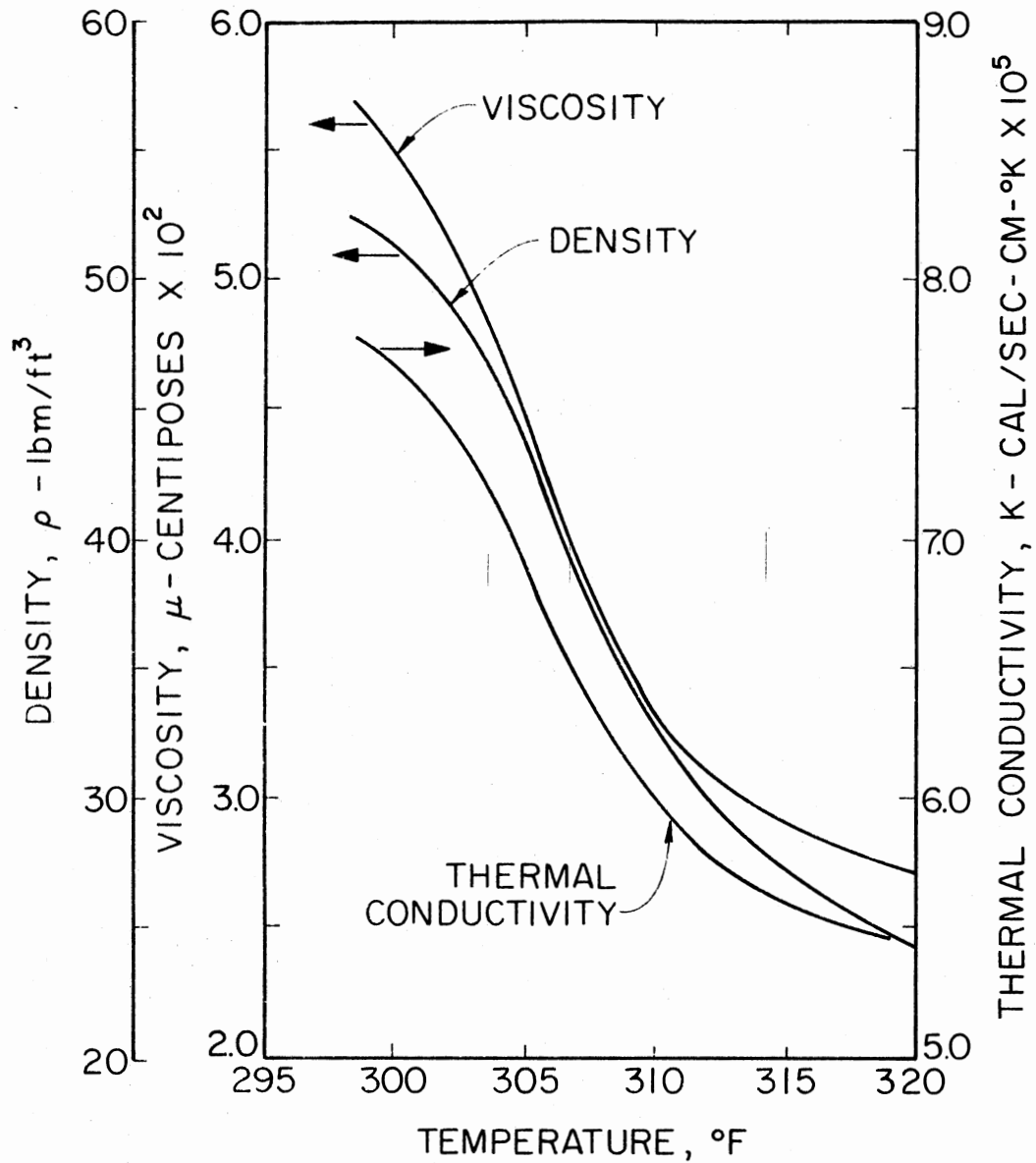


Figure 25. Physical Properties of Refrigerant-114 in the Supercritical Region at 540 psia

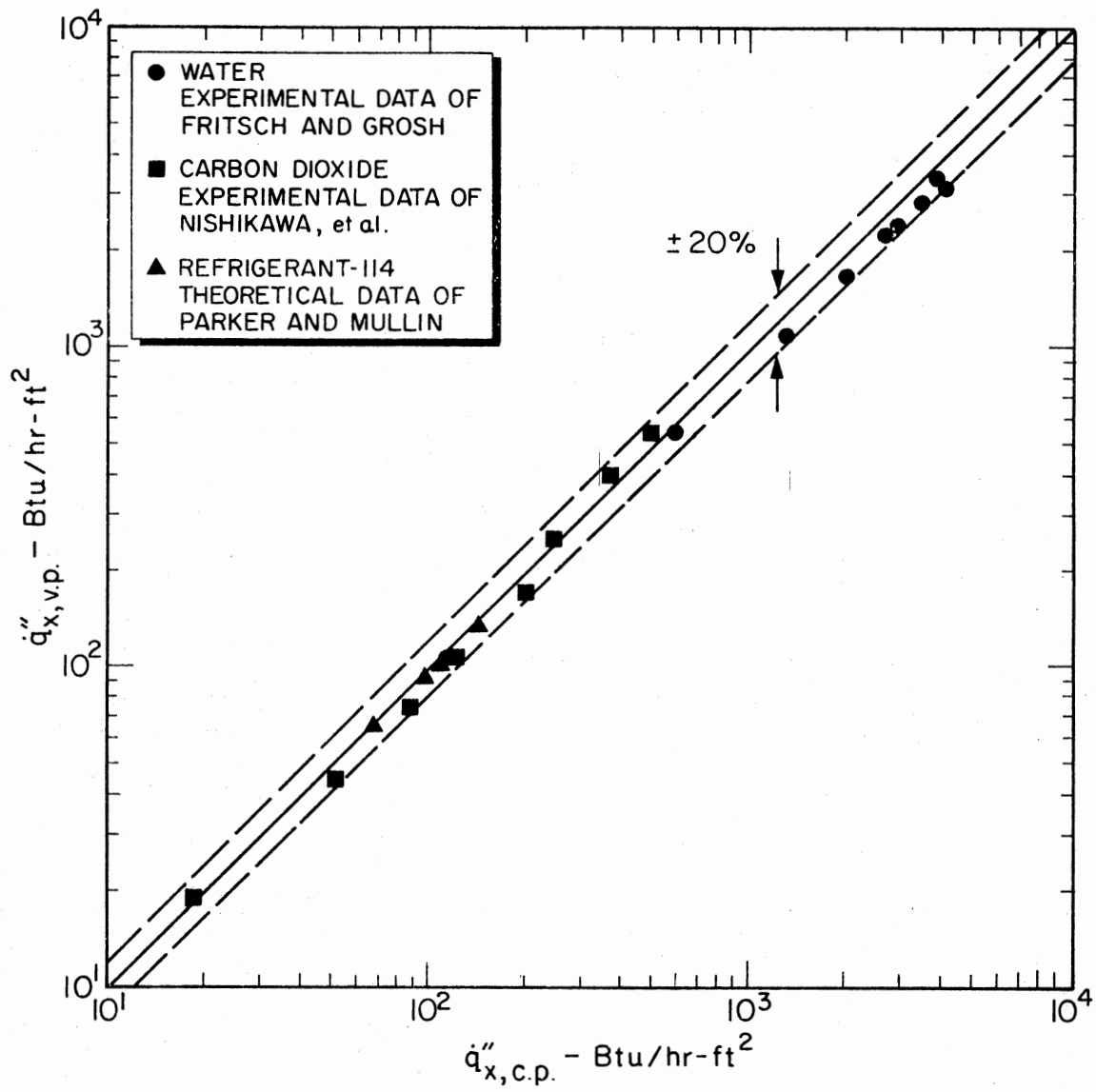


Figure 26. Comparison of Predicted Constant Property Local Heat Flux With Variable Property Local Heat Flux

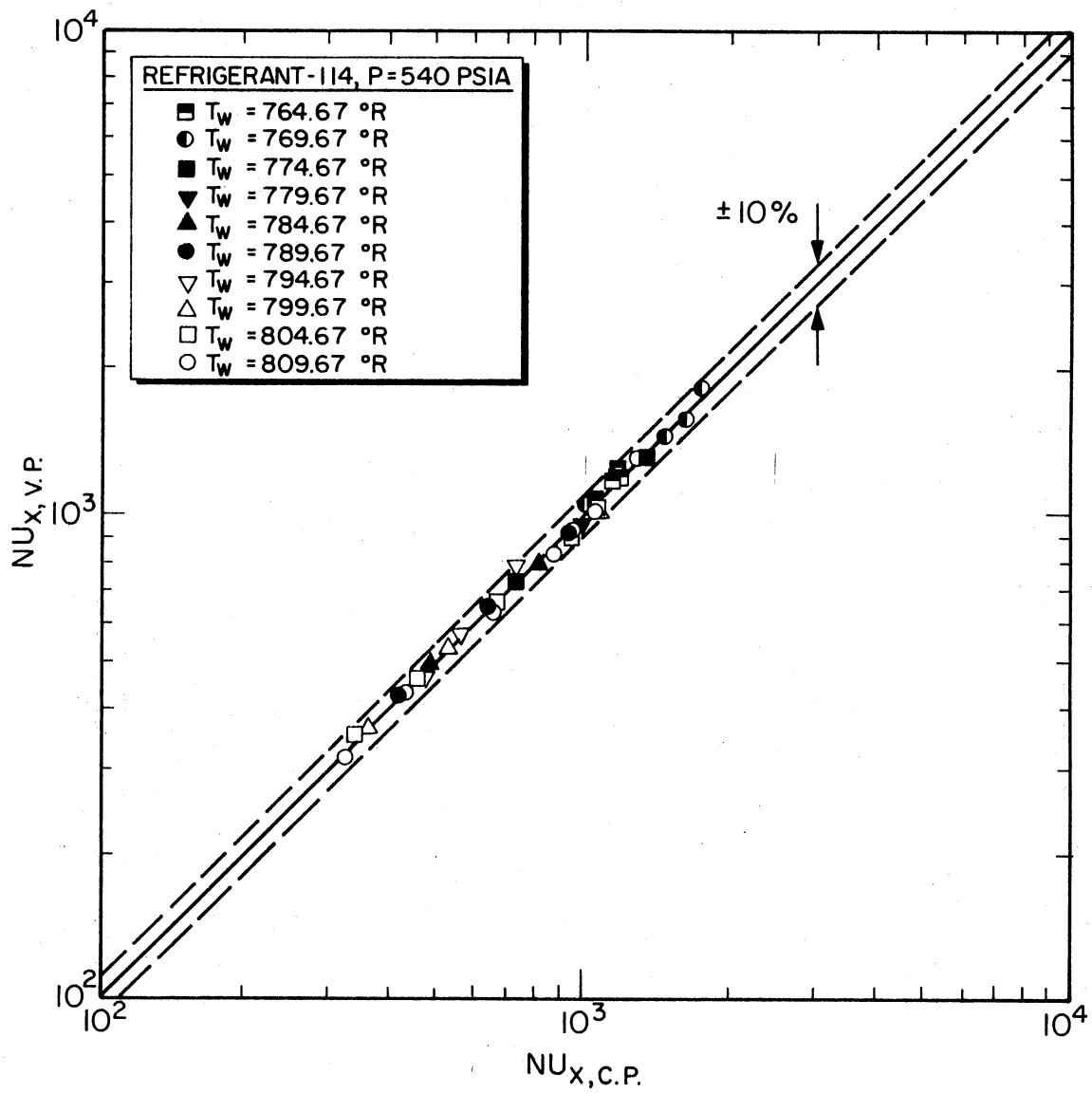


Figure 27. Comparison of Predicted Constant Property Nusselt Numbers With Variable Property Nusselt Numbers

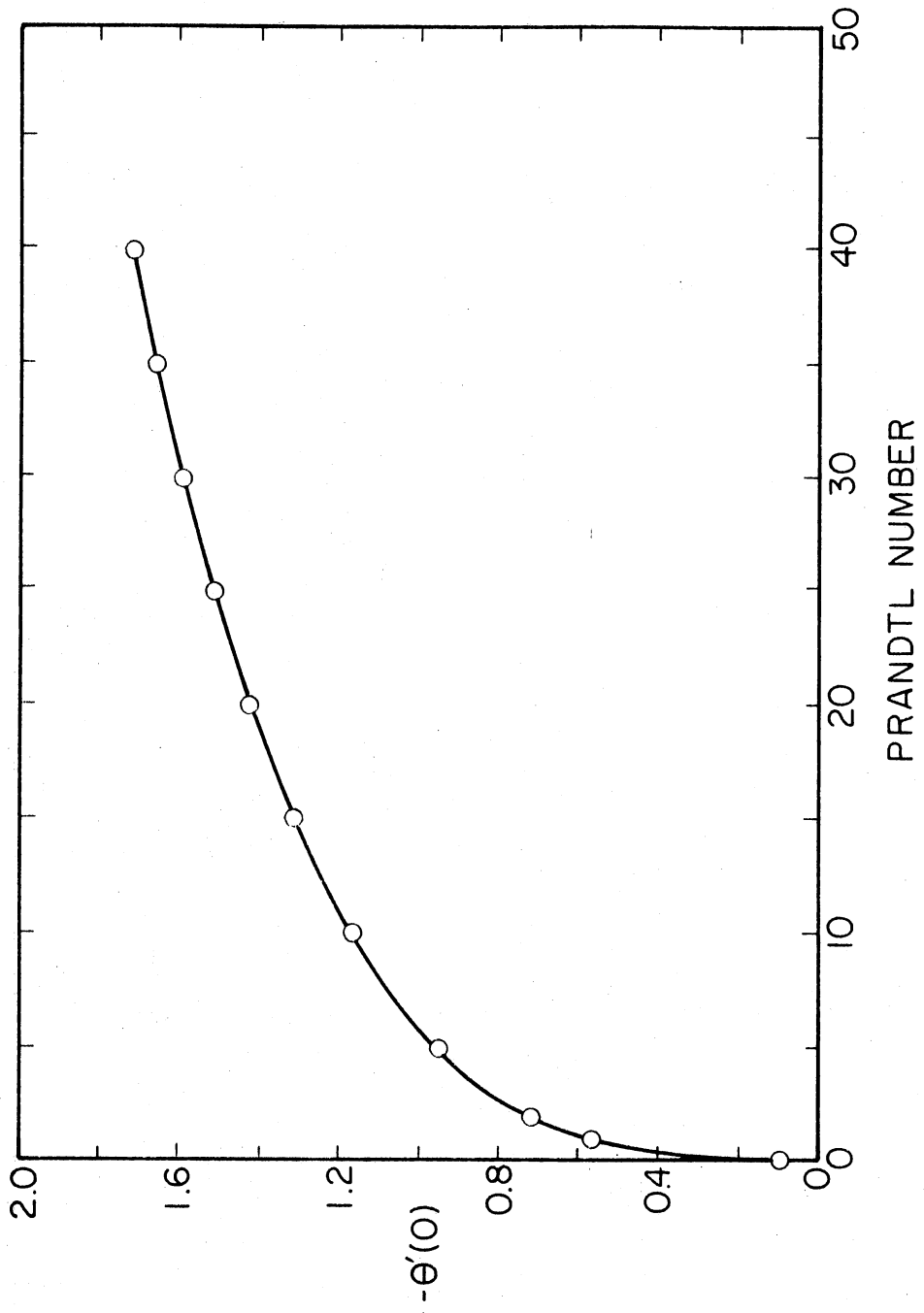


Figure 28. Constant Property Solution

APPENDIX C

EQUATIONS AND DERIVATIONS

Derivation of Equations (3.14) and (3.15)

This section outlines the method by which Equations (3.14) and (3.15) were derived from Equations (3.9) and (3.10). In order to show how this can be accomplished the following is presented:

1. Define the function

$$A = \frac{\rho\mu}{\rho_w \mu_w} = f(\eta) = f(P, T) \quad (C.1)$$

2. The derivative of A with respect to  $\eta$  is

$$\frac{dA}{d\eta} = \frac{\mu}{\rho_w \mu_w} \frac{d\rho}{d\eta} + \frac{\rho}{\rho_w \mu_w} \frac{d\mu}{d\eta} \quad (C.2)$$

3. Note that

$$\rho = \rho(P, T) = f(\eta) \quad (C.3)$$

$$\frac{d\rho}{d\eta} = \left(\frac{\partial\rho}{\partial T}\right)_P \frac{dT}{d\eta} + \left(\frac{\partial\rho}{\partial P}\right)_T \frac{dP}{d\eta} \quad (C.4)$$

4. Since pressure is held constant, this reduces to

$$\frac{d\rho}{d\eta} = \left(\frac{\partial\rho}{\partial T}\right)_P \frac{dT}{d\eta} \quad (C.5)$$

5. The definition of the dimensionless temperature

$$\theta(\eta) = \frac{T - T_\infty}{T_w - T_\infty} \quad (C.6)$$

gives

$$\frac{dT}{d\eta} = (T_w - T_\infty) \theta'(\eta) = \Delta T \theta'(\eta) \quad (C.7)$$

6. The substitution of Equation (C.7) into (C.5) gives

$$\frac{d\rho}{d\eta} = \left(\frac{\partial\rho}{\partial T}\right)_P \Delta T \theta'(\eta) \quad (C.8)$$

7. The same reasoning as above when applied to the viscosity gives

$$\frac{d\mu}{d\eta} = \left(\frac{\partial\mu}{\partial T}\right)_P \Delta T \theta'(\eta) \quad (C.9)$$

8. The substitution of Equations (C.8) and (C.9) into Equation (C.2) yields

$$\frac{dA}{d\eta} = \frac{\mu}{\rho_w \mu_w} \left(\frac{\partial\rho}{\partial T}\right)_P \Delta T \theta'(\eta) + \frac{\rho}{\rho_w \mu_w} \left(\frac{\partial\mu}{\partial T}\right)_P \Delta T \theta'(\eta) \quad (C.10)$$

9. If an identical line of reasoning as the above is applied to the term

$$B = \frac{\rho K}{\rho_w K_w} = f(\eta) = f(P, T) \quad (C.11)$$

the following expression is obtained:

$$\frac{dB}{d\eta} = \frac{K}{\rho_w K_w} \left(\frac{\partial\rho}{\partial T}\right)_P \Delta T \theta'(\eta) + \frac{\rho}{\rho_w \mu_w} \left(\frac{\partial K}{\partial T}\right)_P \Delta T \theta'(\eta) \quad (C.12)$$

10. The substitution of Equations (C.10) and (C.12) into the reduced differential Equations (3.9) and (3.10) yields Equations (3.14) and (3.15).

#### Derivation of Equation (4.11)

This section outlines the method by which Equation (4.11) was obtained based on the equation of state, thermodynamic potential functions, and Maxwell equations. In order to show how this can be accomplished the following is presented:

1. Start with a functional expression for the specific heat at constant volume in terms of temperature and volume, and then form its derivative:

$$C_V = C_V(T, V) \quad (C.13)$$

$$dC_V = \left(\frac{\partial C_V}{\partial T}\right)_V dT + \left(\frac{\partial C_V}{\partial V}\right)_T dV \quad (C.14)$$

2. Using the definition of  $C_V$  and the thermodynamic potential function  $(\partial u / \partial s)_V = T$ , an expression for the second coefficient of Equation (C.14),  $(\partial C_V / \partial V)_T$ , was obtained as follows:

$$C_V = \left(\frac{\partial u}{\partial T}\right)_V = \left(\frac{\partial u}{\partial s}\right)_V \left(\frac{\partial s}{\partial T}\right)_V = T \left(\frac{\partial s}{\partial T}\right)_V \quad (C.15)$$

The derivative of Equation (C.15) was taken with respect to the volume while holding the temperature constant; the result was

$$\left(\frac{\partial C_V}{\partial V}\right)_T = T \left(\frac{\partial^2 s}{\partial V \partial T}\right)_T \quad (C.16)$$

3. Start with the following Maxwell equation

$$\left(\frac{\partial P}{\partial T}\right)_V = \left(\frac{\partial S}{\partial V}\right)_T \quad (C.17)$$

The derivative of the Maxwell Equation (C.17) with respect to volume while holding the temperature constant was obtained as follows:

$$\left(\frac{\partial^2 P}{\partial T^2}\right)_V = \frac{\partial^2 s}{\partial V \partial T} \quad (C.18)$$

4. Equating Equations (C.16) and (C.18) gave



$$\left(\frac{\partial C_V}{\partial V}\right)_T = T \left(\frac{\partial^2 P}{\partial T^2}\right)_V \quad (\text{C.19})$$

5. The substitution of Equation (C.19) into Equation (C.14) and assuming that the temperature was constant gave:

$$dC_V = T \left(\frac{\partial^2 P}{\partial T^2}\right)_V dV \quad (\text{C.20})$$

6. The second derivative of the equation of state, Equation (4.1), with respect to the temperature while holding the volume constant resulted in

$$\left(\frac{\partial^2 P}{\partial T^2}\right)_V = \left(\frac{K}{T_c}\right)^2 \text{EXP}(-K T_r) \left[ \frac{C_2}{(V+b)^2} + \frac{C_3}{(V-b)^3} + \frac{C_5}{(V-b)^5} \right] \quad (\text{C.21})$$

7. The substitution of Equation (C.21) into Equation (C.20) gave

$$\int_{C_{V_0}}^{C_V} dC_V = \int_{V_0=\infty}^V T \left(\frac{K}{T_c}\right)^2 \text{EXP}(-K T_r) \left[ \frac{C_2}{(V-b)^2} + \frac{C_3}{(V-b)^3} + \frac{C_5}{(V-b)^5} \right] dV_T \quad (\text{C.22})$$

8. The integration of Equation (C.22) gave

$$C_V = C_{V_0} - T \left(\frac{K}{T_c}\right)^2 \text{EXP}(-K T_r) \left[ \frac{C_2}{(V-b)} + \frac{C_3}{2(V-b)^2} + \frac{C_5}{4(V-b)^4} \right] \quad (\text{C.23})$$

Equation (C.23), which is the same as Equation (4.11), was used to calculate the effects of pressure change on the specific heat at constant volume for a given temperature. This equation along with Equation (4.9)

was used to evaluate specific heat at constant pressure from Equation (4.10).

### Coefficients of Equations (5.7) Through (5.12)

The expressions given below represent the coefficients of the dimensionless terms A1, 3(A2), 2(A2), A3, B1, and 3(B2) of Equations (5.7) through (5.12).

#### Coefficients for the Term A1

$$A1 = CA + CB \theta + CC \theta^2 + CD \theta^3 \quad (5.7)$$

where

$$CA = (A_{11} + B_{11} T_{\infty} + C_{11} T_{\infty}^2 + D_{11} T_{\infty}^3)(\Delta T)$$

$$CB = (B_{11} + 2C_{11} T_{\infty} + 3D_{11} T_{\infty}^2)(\Delta T)^2$$

$$CC = (C_{11} + 3D_{11} T_{\infty})(\Delta T)^3$$

$$CD = (D_{11})(\Delta T)^4$$

#### Coefficients for the Term 3(A2)

$$3(A2) = CE + CF \theta + CG \theta^2 + CH \theta^3 \quad (5.8)$$

where

$$CE = 3(A_{22} + B_{22} T_{\infty} + C_{22} T_{\infty}^2 + D_{22} T_{\infty}^3)(\rho_w \mu_w)$$

$$CF = 3(B_{22} + 2C_{22} T_{\infty} + 3D_{22} T_{\infty}^2)(\Delta T)(\rho_w \mu_w)$$

$$CG = 3(C_{22} + 3D_{22} T_{\infty})(\Delta T)^2 (\rho_w \mu_w)$$

$$CH = 3(D_{22})(\Delta T)^3 (\rho_w \mu_w)$$

Coefficients for the Term 2(A2)

$$2(A2) = CI + CJ \theta + CK \theta^2 + CL \theta^3 \quad (5.9)$$

where

$$CI = 2(A_{22} + B_{22} T_{\infty} + C_{22} T_{\infty}^2 + D_{22} T_{\infty}^3) (\rho_W \mu_W)$$

$$CJ = 2(B_{22} + 2C_{22} T_{\infty} + 3D_{22} T_{\infty}^2) (\Delta T) (\rho_W \mu_W)$$

$$CK = 2(C_{22} + 3D_{22} T_{\infty}) (\Delta T)^2 (\rho_W \mu_W)$$

$$CL = 2(D_{22}) (\Delta T)^3 (\rho_W \mu_W)$$

Coefficients for the Term A3

$$A3 = (CM + CN \theta + CP \theta^2 + CR \theta^3) - (CS + CT \theta + CU \theta^2 + CW \theta^3) \quad (5.10)$$

where

$$CM = (A_{33} + B_{33} T_{\infty} + C_{33} T_{\infty}^2 + D_{33} T_{\infty}^3) \left( \frac{\rho_W^2 \rho_{\infty} \mu_W}{\rho_{\infty} - \rho_W} \right)$$

$$CN = (B_{33} + 2C_{33} T_{\infty} + 3D_{33} T_{\infty}^2) \left( \frac{\rho_W^2 \rho_{\infty} \mu_W}{\rho_{\infty} - \rho_W} \right) (\Delta T)$$

$$CP = (C_{33} + 3D_{33} T_{\infty}) \left( \frac{\rho_W^2 \rho_{\infty} \mu_W}{\rho_{\infty} - \rho_W} \right) (\Delta T)^2$$

$$CR = (D_{33}) \left( \frac{\rho_W^2 \rho_{\infty} \mu_W}{\rho_{\infty} - \rho_W} \right) (\Delta T)^3$$

$$CS = (A_{22} + B_{22} T_{\infty} + C_{22} T_{\infty}^2 + D_{22} T_{\infty}^3) \left( \frac{\rho_W^2 \mu_W}{\rho_{\infty} - \rho_W} \right)$$

$$CT = (B_{22} + 2C_{22} T_{\infty} + 3D_{22} T_{\infty}^2) \left( \frac{\rho_w^2 \mu_w}{\rho_{\infty} - \rho_w} \right) (\Delta T)$$

$$CU = (C_{22} + 3D_{22} T_{\infty}) \left( \frac{\rho_w^2 \mu_w}{\rho_{\infty} - \rho_w} \right) (\Delta T)^2$$

$$CW = (D_{22}) \left( \frac{\rho_w^2 \mu_w}{\rho_{\infty} - \rho_w} \right) (\Delta T)^3$$

#### Coefficients for the Term B1

$$B1 = CAA + CBB \theta + CCC \theta^2 + CDD \theta^3 \quad (5.11)$$

where

$$CAA = (A_{44} + B_{44} T_{\infty} + C_{44} T_{\infty}^2 + D_{44} T_{\infty}^3) (\Delta T)$$

$$CBB = (B_{44} + 2C_{44} T_{\infty} + 3D_{44} T_{\infty}^2) (\Delta T)^2$$

$$CCC = (C_{44} + 3D_{44} T_{\infty}) (\Delta T)^3$$

$$CDD = (D_{44}) (\Delta T)^4$$

#### Coefficients for the Term 3(B2)

$$3(B2) = CEE + CFF \theta + CGG \theta^2 + CHH \theta^3 \quad (5.12)$$

where

$$CEE = 3(A_{55} + B_{55} T_{\infty} + C_{55} T_{\infty}^2 + D_{55} T_{\infty}^3) (\rho_w \mu_w)$$

$$CFF = 3(B_{55} + 2C_{55} T_{\infty} + 3D_{55} T_{\infty}^2) (\Delta T) (\rho_w \mu_w)$$

$$CGG = 3(C_{55} + 3D_{55} T_{\infty}) (\Delta T)^2 (\rho_w \mu_w)$$

$$CHH = 3(D_{55}) (\Delta T)^3 (\rho_w \mu_w)$$

The terms A1, 3(A2), 2(A2), A3, B1, and 3(B2) expressed by Equations (5.7) through (5.12) are the dimensionless coefficients of the reduced ordinary nonlinear differential equations, Equations (3.14) and (3.15).

Jacobian of the First Order  
Differential Equations

The Jacobians of the first order differential equations, Equation (5.16), are presented here. These Jacobians were presented in the Subroutine JACOB2, which was used in the Lentini-Pereyra Program. The Jacobians of Equation (5.16) were obtained according to

$$\text{JACOB}(I,J) = \frac{\partial(x_I)}{\partial(y_J)}$$

Based on the above equation, the following Jacobians were obtained:

$$\text{JACOB}(1,2) = 1$$

$$\text{JACOB}(2,3) = 1$$

$$\text{JACOB}(3,1) = -(CE + CF G_4 + CG G_4^2 + CH G_4^3) G_3$$

$$\text{JACOB}(3,3) = 2(CI + CJ G_4 + CK G_4^2 + CL G_4^3) G_2$$

$$\begin{aligned} \text{JACOB}(3,3) = & -(CA + CB G_4 + CC G_4^2 + CD G_4^3) G_5 \\ & - (CE + CF G_4 + CG G_4^2 + CH G_4^3) G_1 \end{aligned}$$

$$\begin{aligned} \text{JACOB}(3,4) = & -(CB + 2CC G_4 + 3CD G_4^2) G_5 G_3 \\ & - (CF + 2CG G_4 + 3CH G_4^2) G_1 G_3 \\ & + (CJ + 2CK G_4 + 3CL G_4^2) (G_2)^2 \end{aligned}$$

$$- (CN + 2CP G_4 + 3CR G_4^2)$$

$$+ (CT + 2CU G_4 + 3CW G_4^2)$$

$$JACOB(3,5) = -(CA + CB G_4 + CC G_4^2 + CD G_4^3) G_3$$

$$JACOB(4,5) = 1$$

$$JACOB(5,1) = -(CEE + CFF G_4 + CGG G_4^2 + CHH G_4^3) G_5$$

$$JACOB(5,4) = -(CBB + 2CCC G_4 + 3CDD G_4^2) (G_5)^2$$

$$- (CFF + 2CGG G_4 + 3CHH G_4^2) G_1 G_5$$

$$JACOB(5,5) = -2(CAA + CBB G_4 + CCC G_4^2 + CDD G_4^3) G_5$$

$$- (CEE + CFF G_4 + CGG G_4^2 + CW G_4^3) G_1$$

The Jacobians that are not presented by the above equations were zero.

#### Constant Property Equations for the Lentini-Pereyra Program

The development of the set of first order differential equations and their Jacobians are presented here. These equations were obtained from the constant property reduced differential Equations (3.12) and (3.13).

To obtain the set of first order differential equations, the following procedures were used:

1. Define:

$$G_1 = F \tag{C.24a}$$

$$G_2 = F' \tag{C.24b}$$

$$G_3 = F'' \tag{C.24c}$$

$$G_4 = 0 \quad (C.24d)$$

$$G_5 = 0' \quad (C.24e)$$

2. The set of differential equations, Equations (3.12) and (3.13), become:

$$Y_1 = \frac{dG_1}{d\eta} = G_2 \quad (C.25a)$$

$$Y_2 = \frac{dG_2}{d\eta} = G_3 \quad (C.25b)$$

$$Y_3 = \frac{dG_3}{d\eta} = -3G_1G_3 + 2(G_2)^2 - G_4 \quad (C.25c)$$

$$Y_4 = \frac{dG_4}{d\eta} = G_5 \quad (C.25d)$$

$$Y_5 = \frac{dG_5}{d\eta} = -3Pr G_1G_5 \quad (C.25e)$$

The boundary conditions, Equation (3.11), transform to

$$\left. \begin{array}{l} G_1 = 0 \\ G_2 = 0 \\ G_3 = \text{Unknown} \\ G_4 = 1 \\ G_5 = \text{Unknown} \end{array} \right\} \eta = 0 \quad \left. \begin{array}{l} G_1 = \text{Unknown} \\ G_2 = 0 \\ G_3 = \text{Unknown} \\ G_4 = 0 \\ G_5 = \text{Unknown} \end{array} \right\} \eta = \infty \quad (C.26)$$

Equation (C.25) along with the boundary conditions, Equation (C.26), are the set of first order nonlinear differential equations of the reduced momentum and energy equations for the constant property case to be solved on the computer.

The Jacobians of the first order differential equations, Equation (C.25), were obtained according to

$$\text{JACOB}(I,J) = \frac{\partial(x_I)}{\partial(y_J)}$$

Based on the above equation, the following Jacobians were obtained:

$$\text{JACOB}(1,2) = 1 \quad (\text{C.27a})$$

$$\text{JACOB}(2,3) = 1 \quad (\text{C.27b})$$

$$\text{JACOB}(3,1) = -3G_3 \quad (\text{C.27c})$$

$$\text{JACOB}(3,2) = 4G_2 \quad (\text{C.27d})$$

$$\text{JACOB}(3,3) = -3G_1 \quad (\text{C.27e})$$

$$\text{JACOB}(3,4) = -1 \quad (\text{C.27f})$$

$$\text{JACOB}(4,5) = 1 \quad (\text{C.27g})$$

$$\text{JACOB}(5,1) = -3PrG_5 \quad (\text{C.27h})$$

$$\text{JACOB}(5,5) = -3PrG_1 \quad (\text{C.27i})$$

The Jacobians that are not presented by Equation (C.27) were zero.



APPENDIX D

COMPUTER PROGRAMS

```

SUBROUTINE PROP
*****
* SUBROUTINE PROP *
*****
SUBJECT :
THERMODYNAMIC AND TRANSPORT PROPERTIES OF
WATER IN THE SUPERCRITICAL REGION.
DATA INPUT :
P=PRESSURE(PSIA)
T=TEMPERATURE(DEG.R)
V=SPECIFIC VOLUME (CUFT/LBM)--INITIAL VALUE
DTEMP = TEMP INCREMENT (DEG.R)
NDOLP = MAX NO. OF DOLPOO ITERATION
LIMITATIONS :
REDUCED TEMPERATURE RANGE ( TR = T/TC )
FROM 1.008 TO 1.04 ( WHERE TC = 1165.3 DEG R ).
REDUCED PRESSURE RANGE ( PR = P/PC )
FROM 1.044 TO 1.247 ( WHERE PC = 3208 PSIA ).
IMPLICIT REAL*8(A-H,O-Z)
COMMON/CPROP/DENS(50),CPH(50),VISC(50),COND(50),TEMP(50),DTEMP
COMMON/TERN/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
CONSTANTS FOR THE EQUATION OF STATE.
DATA TC,R/1.16530+03,0.59569247850+00/
DATA BV,H,AA/-0.246705540-01,0.869198010+00,0.719439580-01/
DATA A2,B2,C2/-0.225248090+02,0.116529760-01,-0.702599440+02/
DATA A3,B3,C3/-0.874231150+01,0.706701680-02,-0.199066740+00/
DATA A5,B5,C5/0.121795380-01,-0.15583440-04,0.136796390-01/
CONSTANTS FOR THE SPECIFIC HEAT EQUATION.
DATA X1,X2,X3/-0.236904030-01,-0.229812530+02,0.119774360-01/
DATA X4,X5,X6/-0.72422920+02,0.847504770+00,-0.871769410+01/
DATA X7,X8,X9/0.71550030-02,-0.198038210+00,0.775298840-01/
DATA X10,X11,X12/0.119635170-01,-0.155972590-04,0.129269770-01/
DATA X13,X14,X15/0.114888190+01,0.449630230-04,0.280901340-08/
CONSTANTS FOR THE ABSOLUTE VISCOSITY EQUATION.
DATA AVIS,BVIS,CVIS/0.97170590+01,0.833524050-02,0.102426720+00/
CONSTANTS FOR THE THERMAL CONDUCTIVITY EQUATION.
DATA ACDN,BCDN,DCDN/3.62000+0,0.0380000+2,6.67000/
READ (5,40) P,T,V,DTEMP,NDOLP
40 FORMAT(4F10.4,I4)
COMPUTING THE SPECIFIC VOLUME (CUFT/LBM).
K=1
ITR = 1
Z = V-BV

```

```

PRCP0010
PROP0020
PROP0030
PROP0040
PROP0050
PROP0060
PROP0070
PROP0080
PROP0090
PROP0100
PROP0110
PROP0120
PROP0130
PROP0140
PROP0150
PROP0160
PROP0170
PROP0180
PROP0190
PROP0200
PROP0210
PROP0220
PROP0230
PROP0240
PROP0250
PROP0260
PROP0270
PROP0280
PROP0290
PROP0300
PROP0310
PROP0320
PROP0330
PROP0340
PROP0350
PROP0360
PROP0370
PROP0380
PROP0390
PROP0400
PROP0410
PROP0420
PROP0430
PROP0440
PROP0450
PROP0460
PROP0470
PROP0480
PROP0490
PROP0500
PROP0510
PROP0520
PROP0530
PROP0540
PROP0550
PROP0560
PROP0570
PROP0580
PROP0590

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```

DO 50 J=1,NOCLP,1
TEMP(J)=T
TR = T/TC
EXPNEG = (1.00+00)/(DEXPH*TR)
CDEFF*S FOR 5TH DEGREE POLY.
A00 = P
A11 = R*T
A22 = (A2)+(B2*T)+(C2*EXPNEG)
A33 = (A3)+(B3*T)+(C3*EXPNEG)
A44 = (A4)
A55 = (A5)+(B5*T)+(C5*EXPNEG)
NEWTON RAPHSON ITERATION.
43 FM = (((A00+Z*A11)+Z-A22)+Z-A33)+Z-A44)+Z-A55
FNDR = ((45.00+00*A00+Z-4.00+00*A11)+Z-3.00+00*A22)+Z
1-2.00+00*A33)+Z-AA4
ZNEW = Z - (FM/FNDR)
ZDIFF = ZNEW - Z
VZ=ZNEW+BV
IF(VZ.LE. 0.0001) GO TO 31
IF(DABS(ZDIFF).LE. 1.00D-5) GO TO 45
31 Z = ZNEW
ITR = ITR + 1
IF(ITR.GT.300) GO TO 44
GO TO 43
44 WRITE(6,30) P,T
30 FORMAT(1H0,5X,"MAXIMUM ITERATION REACHED FOR SPECIFIC VOLUME AT",
5X," P = ",F10.5," AND T = ",F10.5)
45 SPVOL = ZNEW + BV
COMPUTING THE DENSITY (LBM/CUFT).
DENSJ = (1.00+00)/SPVOL
DENS(J)=DENSJ
COMPUTING THE DERIVATIVES OF SPECIFIC VOLUME AND DENSITY.
F11 = R/ZNEW
F22 = ((B2)-(H/TC)+(C2)*(EXPNEG))/(ZNEW**2.0)
F33 = ((B3)-(H/TC)+(C3)*(EXPNEG))/(ZNEW**3.0)
F55 = ((B5)-(H/TC)+(C5)*(EXPNEG))/(ZNEW**5.0)
F66 = (A11)/(ZNEW**2.0)
F77 = ((2.00+00)*(A22))/(ZNEW**3.0)
F88 = ((3.00+00)*(A33))/(ZNEW**4.0)
F99 = ((4.00+00)*(A44))/(ZNEW**5.0)
F10 = ((5.00+00)*(A55))/(ZNEW**6.0)
DERPTV = (F11)+(F22)+(F33)+(F55)
DERPVT = -(F66)-(F77)-(F88)-(F99)-(F10)
DERVTP = -(DERPTV/DERPVT)
DEROTP = -(DENSJ**2.0)*(DERVTP)
COMPUTING THE SPECIFIC HEAT AT CONSTANT PRESSURE (BTU/LBM*DEGR).
ZCP=SPVOL-X1
EXPCP=(1.000)/(DEXPH*(X5*TR))
D0=P
D1=R*T
D2=X2+(X3*T)+(X4*EXPCP)
D3=X6+(X7*T)+(X8*EXPCP)
D4=X9
D5=X10+(X11*T)+(X12*EXPCP)

```

```

PRCP0600
PROP0610
PROP0620
PROP0630
PROP0640
PROP0650
PROP0660
PROP0670
PRCP0680
PROP0690
PROP0700
PROP0710
PROP0720
PROP0730
PROP0740
PROP0750
PRCP0760
PROP0770
PROP0780
PROP0790
PROP0800
PROP0810
PROP0820
PROP0830
PROP0840
PROP0850
PROP0860
PROP0870
PROP0880
PRCP0890
PROP0900
PROP0910
PROP0920
PROP0930
PROP0940
PROP0950
PROP0960
PROP0970
PROP0980
PROP0990
PROP1000
PRCP1010
PROP1020
PROP1030
PRCP1040
PROP1050
PROP1060
PROP1070
PROP1080
PROP1090
PROP1100
PROP1110
PROP1120
PROP1130
PROP1140
PROP1150
PROP1160
PROP1170
PROP1180
PROP1190

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```

53 FNCP=(((D0+ZCP-D1)*ZCP-D2)+ZCP-D3)*ZCP-D4)+ZCP-D5      PROP1200
FNDR=(((S.000+D0+ZCP-4.000+D1)*ZCP-3.000+D2)*ZCP-2.000+D3)+ZCP-D4 PROP1210
ZNCP=ZCP-(FNCP/FNDR)                                       PROP1220
ZOIF=ZNCP-ZCP                                             PROP1230
VZCP=ZNCP+X1                                              PROP1240
IF(VZCP .LE. 0.000) GO TO 35                               PROP1250
IF(DABS(ZOIF) .LE. 1.00-05 ) GO TO 55                     PROP1260
15 ZCP=ZNCP                                                PROP1270
K=K+1                                                       PROP1280
IF(K .GT. 900 ) GO TO 11                                  PROP1290
GO TO 53                                                    PROP1300
11 WRITE(6,21) P,T                                         PROP1310
21 FORMAT(1H0,5X,'MAXIMUM ITERATION REACHED FOR SPECIFIC HEAT AT ',//PROP1320
5X,' P = ',F10.5,' AND T = ',F10.5)
55 CONTINUE                                                PROP1330
F111=R/ZNCP                                                PROP1340
F222=(((X3)-(X5/TC)*(X4)*(EXPCP)))/(ZNCP**2)             PROP1350
F333=(((X7)-(X5/TC)*(X8)*(EXPCP)))/(ZNCP**3)             PROP1360
F555=(((X11)-(X5/TC)*(X12)*(EXPCP)))/(ZNCP**5)           PROP1370
F666=(D1)/(ZNCP**2)                                        PROP1380
F777=(((2.00)*(D2))/(ZNCP**3)                             PROP1390
F888=(((3.00)*(D3))/(ZNCP**4)                             PROP1400
F999=(((4.00)*(D4))/(ZNCP**5)                             PROP1410
F100=(((5.00)*(D5))/(ZNCP**6)                             PROP1420
DRPTV=((F111)+(F222)*((F333)+(F555)                       PROP1440
DRPVT=-((F666)-(F777)-(F888)-(F999)-(F100)               PROP1450
DRVTP=-((DRPTV/DRPVT)                                     PROP1460
CVLP=X13+(X14*T)+(X15*T**2)                                PROP1470
W22=Te((X5/TC)**2)*(X4*(EXPCP))/(ZNCP)                   PROP1480
W33=Te((X5/TC)**2)*(X8*(EXPCP))/(2.00*(ZNCP**2))         PROP1490
W55=Te((X5/TC)**2)*(X12*(EXPCP))/(4.00*(ZNCP**4))       PROP1500
CVHP=(CVLP)-((W22+W33+W55)*(1.44+02/7.780+02))          PROP1510
CPHP=(CVHP)+Te((DRPTV)/(DRPVT))*(1.44+02/7.780+02)      PROP1520
CPH(J)=CPHP                                               PROP1530
C                                                           PROP1540
C COMPUTING THE ABSOLUTE VISCOSITY (CENTIPOISE).          PROP1550
C                                                           PROP1560
C VISHP=(AVIS+(BVIS*T)+(CVIS+DENSTY**1.57))*(1.00-06/0.000672000) PROP1570
C VISC(J)=VISHP                                           PROP1580
C                                                           PROP1590
C COMPUTING THE DERIVATIVES OF THE ABSOLUTE VISCOSITY EQUATION. PROP1600
C                                                           PROP1610
C DERUDT=(CVIS+1.00-06+1.57D0*DENSTY**0.57)/(0.000672000) PROP1620
C DERUTD=(BVIS+1.00-06)/(0.000672000)                    PROP1630
C DERUTP=(DERUDT)*(DERDTP)+(DERUTD)                       PROP1640
C                                                           PROP1650
C COMPUTING THE THERMAL CONDUCTIVITY (BTU/FT-HR-DEG R).  PROP1660
C                                                           PROP1670
C TF=T-459.67D00                                           PROP1680
C CONHP=(ACON+(BCON*TF)+(DCON+DENSTY**1.24))*1.00-03    PROP1690
C COND(J)=CONHP                                           PROP1700
C                                                           PROP1710
C COMPUTING THE DERIVATIVES OF THE THERMAL CONDUCTIVITY EQUATION. PROP1720
C                                                           PROP1730
C DERKDT=1.24D00+DCON+1.00-03+DENSTY**0.24              PROP1740
C DERKTD=BCON+1.00-03                                     PROP1750
C DERKTP=(DERKDT)*(DERDTP)+(DERKTD)                       PROP1760
C                                                           PROP1770
C COMPUTING THE COEFF'S OF THE DIFF. EQUATIONS.          PROP1780
C                                                           PROP1790
DCOEFF=(DERDTP)/(DENSTY)                                  PROP1800
ECOEFF=(DERUTP)/(VISHP)                                   PROP1810
HCOEFF=(DERKTP)/(CONHP)                                  PROP1820
PRND=(CPMP+VISHP*2.42D00)/(CONHP)                       PROP1830
C                                                           PROP1840
C AA1(J)=DCOEFF+ECOEFF                                     PROP1850
C AA2(J)=1.00+00/(DENSTY*VISHP)                          PROP1860
C AA3(J)=1.00+00/(DENSTY*DENSTY*VISHP)                   PROP1870
C BB1(J)=DCOEFF+HCOEFF                                    PROP1880
C BB2(J)=-(CPMP)/(DENSTY*CONHP)                          PROP1890
C                                                           PROP1900
C T=T+DTEMP                                               PROP1910
C 50 ITR=1                                                 PROP1920
C                                                           PROP1930
C RETURN                                                  PROP1940
C END                                                       PROP1950

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SUBROUTINE PROP
*****
SUBROUTINE PROP
*****
SUBJECT : THERMODYNAMIC AND TRANSPORT PROPERTIES OF
          CARBON DIOXIDE IN THE SUPERCRITICAL REGION.
DATA INPUT :
P=PRESSURE(P.SIA)
T=TEMPERATURE(DEG.R)
V=SPECIFIC VOLUME (CUFT/LBM)--INITIAL VALUE
DTEMP = TEMP INCREMENT (DEG.R)
NDOLP = MAX NO. OF DOLoop ITERATION
LIMITATIONS :
REDUCED TEMPERATURE RANGE ( TR = T/TC )
FROM 1.0028 TO 1.1455 ( WHERE TC = 547.56 DEG R ).
REDUCED PRESSURE RANGE ( PR = P/PC )
FROM 1.056 TO 1.955 ( WHERE PC = 1070 PSIA ).
IMPLICIT REAL*8(A-H,O-Z)
COMMON/CPROP/DENS(50),CPH(50),VISC(50),COND(50),TEMP(50),DTEMP
COMMON/TERM/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
CONSTANTS FOR THE EQUATION OF STATE.
DATA TC,R/547.56000,0.2438400364000/
DATA BV,H,AA4/-0.18588948D-01,0.12018118D+01,0.18744415D-01/
DATA A2,B2,C2/-0.13326704D+02,-0.83298325D-02,0.62417313D+02/
DATA A3,B3,C3/-0.73090008D+01,0.90982283D-02,0.57368866D+01/
DATA A5,B5,C5/0.1095804D-01,-0.1265501D-04,-0.13531694D-01/
CONSTANTS FOR THE SPECIFIC HEAT EQUATION.
DATA X1,X2,X3/-0.30146832D-01,-0.10972492D+02,-0.56307166D-02/
DATA X4,X5,X6/0.47571869D+02,0.12092998D+01,-0.72139233D+01/
DATA X7,X8,X9/0.90332088D-02,0.63052626D+01,0.13941827D-02/
DATA X10,X11,X12/0.11295915D-01,-0.12727667D-04,-0.13171462D-01/
DATA X13,X14,X15/0.70600271D+01,-0.17494759D-01,0.10779554D-04/
CONSTANTS FOR THE ABSOLUTE VISCOSITY EQUATION.
DATA AVIS,BVIS/-0.20033880D+01,0.22677910D-01/
DATA CVIS,DVIS/0.10305940D-01,0.20890480D+01/
CONSTANTS FOR THE THERMAL CONDUCTIVITY EQUATION.
DATA ACON,BCON/0.84746444D+01,-0.44669005D-01/
DATA DCON,ECOM/0.11056622D+02,0.46278907D+00/
READ (5,40) P,T,V,DTEMP,NDOLP
40 FORMAT(4F10.4,I4)
COMPUTING THE SPECIFIC VOLUME (CUFT/LBM).
K=1

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```

PRCP0010
PRCP0020
PRCP0030
PRCP0040
PRCP0050
PRCP0060
PRCP0070
PRCP0080
PRCP0090
PRCP0100
PRCP0110
PRCP0120
PRCP0130
PRCP0140
PRCP0150
PRCP0160
PRCP0170
PRCP0180
PRCP0190
PRCP0200
PRCP0210
PRCP0220
PRCP0230
PRCP0240
PRCP0250
PRCP0260
PRCP0270
PRCP0280
PRCP0290
PRCP0300
PRCP0310
PRCP0320
PRCP0330
PRCP0340
PRCP0350
PRCP0360
PRCP0370
PRCP0380
PRCP0390
PRCP0400
PRCP0410
PRCP0420
PRCP0430
PRCP0440
PRCP0450
PRCP0460
PRCP0470
PRCP0480
PRCP0490
PRCP0500
PRCP0510
PRCP0520
PRCP0530
PRCP0540
PRCP0550
PRCP0560
PRCP0570
PRCP0580
PRCP0590

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```

ITR = 1
Z = V-BV
DO 50 J=1,NDOLP,1
TEMPIJ)=T
TR = T/TC
EXPNEG = (1.0D+00)/(DEXP(H*TR))
COEFF'S FOR 5TH DEGREE POLY.
A00 = P
A11 = R*T
A22 = (A2)+(B2*TR)+(C2*EXPNEG)
A33 = (A3)+(B3*TR)+(C3*EXPNEG)
A44 = (A4)
A55 = (A5)+(B5*TR)+(C5*EXPNEG)
NEWTON RAPHSON ITERATION.
43 FN = (((A00*Z-A11)*Z-A22)*Z-A33)*Z-A44)*Z-A55
FNDER = (((5.0D+00*A00*Z-4.0D+00*A11)*Z-3.0D+00*A22)*Z
1-2.0D+00*A33)*Z-A44
ZNEW = Z - (FN/FNDER)
ZDIFF = ZNEW - Z
VZ=ZNEW*BV
IF(VZ .LE. 0.000) GO TO 31
IF(DABS(ZDIFF).LE. 1.0D-5) GO TO 45
31 Z = ZNEW
ITR = ITR + 1
IF(ITR .GT.300) GO TO 44
GO TO 43
44 WRITE(6,30) P,T
30 FORMAT(1H0,5X,'MAXIMUM ITERATION REACHED FOR SPECIFIC VOLUME AT',
5X,' P = ',F10.5,' AND T = ',F10.5)
45 SPVOL = ZNEW + BV
COMPUTING THE DENSITY (LBM/CUFT).
DENSY = (1.0D+00)/SPVOL
DENS(IJ)=DENSY
COMPUTING THE DERIVATIVES OF SPECIFIC VOLUME AND DENSITY.
F11 = R/ZNEW
F22 = ((B2)-(H/TC))*(C2)*(EXPNEG)/(ZNEW**2.0)
F33 = ((B3)-(H/TC))*(C3)*(EXPNEG)/(ZNEW**3.0)
F55 = ((B5)-(H/TC))*(C5)*(EXPNEG)/(ZNEW**5.0)
F66 = (A11)/(ZNEW**2.0)
F77 = ((2.0D+00)*(A22))/(ZNEW**3.0)
F88 = ((3.0D+00)*(A33))/(ZNEW**4.0)
F99 = ((4.0D+00)*(A44))/(ZNEW**5.0)
F10 = ((5.0D+00)*(A55))/(ZNEW**6.0)
DERPVT = (F11)+(F22)+(F33)+(F55)
DERPVT = -(F66)-(F77)-(F88)-(F99)-(F10)
DERVTP = -(DERPVT/DERPVT)
DERDTP = -(DENSY**2.0)*(DERVTP)
COMPUTING THE SPECIFIC HEAT AT CONSTANT PRESSURE (BTU/LBM*DEGR).
ZCP=SPVOL-X1
D0=P
EXPCP=(1.0D0)/(DEXP(X5*TR))
D1=R*T
D2=X2+(X3*TR)+(X4*EXPCP)
D3=X6+(X7*TR)+(X8*EXPCP)
PRCP0600
PRCP0610
PRCP0620
PRCP0630
PRCP0640
PRCP0650
PRCP0660
PRCP0670
PRCP0680
PRCP0690
PRCP0700
PRCP0710
PRCP0720
PRCP0730
PRCP0740
PRCP0750
PRCP0760
PRCP0770
PRCP0780
PRCP0790
PRCP0800
PRCP0810
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PRCP0830
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PRCP0850
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PRCP0870
PRCP0880
PRCP0890
PRCP0900
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PRCP0920
PRCP0930
PRCP0940
PRCP0950
PRCP0960
PRCP0970
PRCP0980
PRCP0990
PRCP1000
PRCP1010
PRCP1020
PRCP1030
PRCP1040
PRCP1050
PRCP1060
PRCP1070
PRCP1080
PRCP1090
PRCP1100
PRCP1110
PRCP1120
PRCP1130
PRCP1140
PRCP1150
PRCP1160
PRCP1170
PRCP1180
PRCP1190

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D4=X9
D5=X10+(X11*T)+(X12*EXPCP)
53 FNCP=(((D0*ZCP-D1)*ZCP-D2)*ZCP-D3)*ZCP-D4)*ZCP-D5
FNDR=(((5.00*00*ZCP-4.00*01)*ZCP-3.00*02)*ZCP-2.00*03)*ZCP-04
ZNCP=ZCP-(FNCP/FNDR)
ZDIF=ZNCP-ZCP
VZCP=ZNCP*X1
IF(VZCP .LE. 0.000) GO TO 35
IF(DABS(ZDIF) .LE. 1.0D-05 ) GO TO 55
35 ZCP=ZNCP
K=K+1
IF(K .GT. 900 ) GO TO 11
GO TO 53
11 WRITE(6,21) P,T
21 FORMAT(1H0,5X,'MAXIMUM ITERATION REACHED FOR SPECIFIC HEAT AT ',//
*5X,* P = ',F10.5,' AND T = ',F10.5)
55 CONTINUE
F111=R/ZNCP
F222=((X3)-(X5/TC)*(X4)*(EXPCP))/(ZNCP**2)
F333=((X7)-(X5/TC)*(X8)*(EXPCP))/(ZNCP**3)
F555=((X11)-(X5/TC)*(X12)*(EXPCP))/(ZNCP**5)
F666=(D1)/(ZNCP**2)
F777=((2.00)*(D2))/(ZNCP**3)
F888=((3.00)*(D3))/(ZNCP**4)
F999=((4.00)*(D4))/(ZNCP**5)
F100=((5.00)*(D5))/(ZNCP**6)
DRPTV=(F111)+(F222)+(F333)+(F555)
DRPVT=-(F666)-(F777)-(F888)-(F999)-(F100)
DRVTP=-(DRPTV/DRPVT)
CVLP=X13+(X14*T)+(X15*T**2)
W22=T*((X5/TC)**2)*(X4*(EXPCP))/(ZNCP)
W33=T*((X5/TC)**2)*(X8*(EXPCP))/(2.00*(ZNCP**2))
W55=T*((X5/TC)**2)*(X12*(EXPCP))/(4.00*(ZNCP**4))
CVHP = (CVLP) - ((W22+W33+W55)*(1.44D+02/7.78D+02))
CPHP=(CVHP)+T*(DRPTV)*(DRVTP)*(1.44D+02/7.78D+02)
CPH(J)=CPHP
C
C COMPUTING THE ABSOLUTE VISCOSITY (CENTIPoise).
C
VISHP=(AVIS+(BVIS*T)+(CVIS*DENSTY**DVIS))*(1.0D-06/0.000672D00)
VISC(J)=VISHP
C
C COMPUTING THE DERIVATIVES OF THE ABSOLUTE VISCOSITY EQUATION.
C
DERUDT=(CVIS*1.0D-06*DVIS*DENSTY**((DVIS-1.0D00))/(0.000672D00)
DERUTD=BVIS*1.0D-06/(0.000672D00)
DERUTP=(DERUDT)*(DERDTP)+(DERUTD)
C
C COMPUTING THE THERMAL CONDUCTIVITY (BTU/FT-HR-DEG R).
C
CONHP=(ACON+(BCON*T)+(DCON*DENSTY**ECON))*1.0D-03
COND(J)=CONHP
C
C COMPUTING THE DERIVATIVES OF THE THERMAL CONDUCTIVITY EQUATION.
C
DERKDT=DCON*ECON*1.0D-03*DENSTY**((ECON-1.0D00)
DERKTD=BCON*1.0D-03
DERKTP=(DERKDT)*(DERDTP)+(DERKTD)
C
C COMPUTING THE COEFF'S OF THE DIFF. EQUATIONS.

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PROP1200
PROP1210
PROP1220
PROP1230
PROP1240
PROP1250
PROP1260
PROP1270
PROP1280
PROP1290
PROP1300
PROP1310
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PROP1330
PROP1340
PROP1350
PROP1360
PROP1370
PROP1380
PROP1390
PROP1400
PROP1410
PROP1420
PROP1430
PROP1440
PROP1450
PROP1460
PROP1470
PROP1480
PROP1490
PROP1500
PROP1510
PROP1520
PROP1530
PROP1540
PROP1550
PROP1560
PROP1570
PROP1580
PROP1590
PROP1600
PROP1610
PROP1620
PROP1630
PROP1640
PROP1650
PROP1660
PROP1670
PROP1680
PROP1690
PROP1700
PROP1710
PROP1720
PROP1730
PROP1740
PROP1750
PROP1760
PROP1770
PROP1780
PROP1790

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C
DCOEFF = (DERDTP)/(DENSTY)
ECCEFF = (DERUTP)/(VISHP)
HCCEFF = (DERKTP)/(CONHP)
PRNO = (CPHP*VISHP*2.42000)/(CONHP)
C
AA1(J)=DCOEFF + ECCEFF
AA2(J)=1.0D+00/(DENSTY*VISHP)
AA3(J)=1.0D+00/(DENSTY*DENSTY*VISHP)
BB1(J)=DCOEFF + HCCEFF
BB2(J)=(CPHP)/(DENSTY*CONHP)
C
T=T+DTEMP
50 ITR=1
C
RETURN
END
PROP1800
PROP1810
PROP1820
PROP1830
PROP1840
PROP1850
PROP1860
PROP1870
PROP1880
PROP1890
PROP1900
PROP1910
PROP1920
PROP1930
PROP1940
PROP1950
PROP1960

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SUBROUTINE PRCP
*****
* SUBROUTINE PROP *
*****
SUBJECT :
THERMODYNAMIC AND TRANSPORT PROPERTIES OF
REFRIGERANT 114 IN THE SUPERCRITICAL REGION.
DATA INPUT :
P=PRESSURE(P.SIA)
T=TEMPERATURE(DEG.R)
V=SPECIFIC VOLUME (CUFT/LBM)--INITIAL VALUE
DTEMP = TEMP INCREMENT (DEG.R)
NDOLP = MAX NO. OF DOLOOP ITERATION
LIMITATIONS :
REDUCED TEMPERATURE RANGE ( TR = T/TC )
FROM 1.0076 TO 1.074 ( WHERE TC = 753.95 DEG R ).
REDUCED PRESSURE RANGE ( PR = P/PC )
FROM 1.0638 TO 1.2766 ( WHERE PC = 473 PSIA ).
IMPLICIT REAL*8(A-H,O-Z)
COMMON/CPROP/DENS(50),CPH(50),VISC(50),COND(50),TEMP(50),DTEMP
COMMON/TERM/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
CONSTANTS FOR THE EQUATION OF STATE.
DATA TC,R/7.5395D+02,0.06280187D00/
DATA BV,H,AA/5.9149070D-03,3.0D+00,-3.8574810D-04/
DATA A2,B2,C2/2.3856704D+00,1.0801207D-03,-6.5643648D+00/
DATA A3,B3,C3/3.4055687D-02,-5.3336494D-06,1.6366057D-01/
DATA A5,B5,C5/1.6017659D-06,6.2632340D-10,-1.0165314D-05/
CONSTANTS FOR THE SPECIFIC HEAT EQUATION.
SEE THE APPROPRIATE EQUATIONS IN THE PROGRAM.
CONSTANTS FOR THE ABSOLUTE VISCOSITY EQUATION.
SEE THE APPROPRIATE EQUATIONS IN THE PROGRAM.
CONSTANTS FOR THE THERMAL CONDUCTIVITY EQUATION.
SEE THE APPROPRIATE EQUATIONS IN THE PROGRAM.
READ (5,40) P,T,V,DTEMP,NDOLP
40 FORMAT(4F10.4,I4)
COMPUTING THE SPECIFIC VOLUME (CUFT/LBM).
K=1
ITR = 1
Z = V-BV
DO 50 J=1,NDOLP,1
TEMP(J)=T
TR = T/TC
EXPNEG = (1.0D+00)/(DEXP(H*TR))

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PRCP0010
PRCP0020
PRCP0030
PRCP0040
PRCP0050
PRCP0060
PRCP0070
PRCP0080
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PRCP0100
PRCP0110
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PRCP0140
PRCP0150
PRCP0160
PRCP0170
PRCP0180
PRCP0190
PRCP0200
PRCP0210
PRCP0220
PRCP0230
PRCP0240
PRCP0250
PRCP0260
PRCP0270
PRCP0280
PRCP0290
PRCP0300
PRCP0310
PRCP0320
PRCP0330
PRCP0340
PRCP0350
PRCP0360
PRCP0370
PRCP0380
PRCP0390
PRCP0400
PRCP0410
PRCP0420
PRCP0430
PRCP0440
PRCP0450
PRCP0460
PRCP0470
PRCP0480
PRCP0490
PRCP0500
PRCP0510
PRCP0520
PRCP0530
PRCP0540
PRCP0550
PRCP0560
PRCP0570
PRCP0580
PRCP0590

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C COEFF'S FOR 5TH DEGREE POLY.
AC0 = P
A11 = R*T
A22 = (A2)+(B2*T)+(C2*EXPNEG)
A33 = (A3)+(B3*T)+(C3*EXPNEG)
A44 = (A4)
A55 = (A5)+(B5*T)+(C5*EXPNEG)
C NEWTON RAPHSON ITERATION.
43 FN = (((100*T-A11)*Z-A22)*Z-A33)*Z-A44)*Z-A55
FNDER = (((5.0D+00*A00*Z-4.0D+00*A11)*Z-3.0D+00*A22)*Z
1-2.0D+00*A33)*Z-A44
ZNEW = Z - (FN/FNDER)
ZDIFF = ZNEW - Z
VZ=ZNEW+BV
IF (VZ .LE. 0.0D0) GO TO 31
IF (DABS(ZDIFF) .LE. 1.0D-5) GO TO 45
31 Z = ZNEW
ITR = ITR + 1
IF (ITR .GT. 100) GO TO 44
GO TO 43
44 WRITE(6,30) P,T
30 FORMAT(1H0,5X,'MAXIMUM ITERATION REACHED FOR SPECIFIC VOLUME AT',
*5X,' P = ',F10.5,' AND T = ',F10.5)
45 SPVOL = ZNEW + BV
C COMPUTING THE DENSITY (LBM/CUFT).
DENSY = (1.0D+00)/SPVOL
DENS(J)=DENSY
C COMPUTING THE DERIVATIVES OF SPECIFIC VOLUME AND DENSITY.
F11 = R/ZNEW
F22 = ((B2)-(H/TC)*(C2)*(EXPNEG))/(ZNEW**2.0)
F33 = ((B3)-(H/TC)*(C3)*(EXPNEG))/(ZNEW**3.0)
F55 = ((B5)-(H/TC)*(C5)*(EXPNEG))/(ZNEW**5.0)
F66 = (A11)/(ZNEW**2.0)
F77 = ((2.0D+00)*(A22))/(ZNEW**3.0)
F88 = ((3.0D+00)*(A33))/(ZNEW**4.0)
F99 = ((4.0D+00)*(A44))/(ZNEW**5.0)
F10 = ((5.0D+00)*(A55))/(ZNEW**6.0)
DERPTV = (F11)+(F22)+(F33)+(F55)
DERPVT = -(F66)-(F77)-(F88)-(F99)-(F10)
DERVTP = -(DERPTV/DERPVT)
DERDTP = -(DENSY**2.0)*(DERVTP)
C COMPUTING THE SPECIFIC HEAT AT CONSTANT VOLUME (BTU/LBM*DEGR).
CVLP = (1.75D-02)+(3.49D-04)*(T)-(1.67D-07)*(T**2.0)
W22 = ((T)*(H/TC)**2.0)*(C2)*(EXPNEG)/(ZNEW)
W33 = ((T)*(H/TC)**2.0)*(C3)*(EXPNEG)/(2.0D+00*(ZNEW**2.0))
W55 = ((T)*(H/TC)**2.0)*(C5)*(EXPNEG)/(4.0D+00*(ZNEW**4.0))
CVHP = (CVLP) - ((W22+W33+W55)*(1.44D+02/7.78D+02))
C COMPUTING THE SPECIFIC HEAT AT CONSTANT PRESSURE (BTU/LBM*DEGR).
CPHP = (CVHP)+(T)*(DERPTV)*(DERVTP)*(1.44D+02/7.78D+02)
CPH(J)=CPHP
C COMPUTING THE ABSOLUTE VISCOSITY (CENTIPOISE).

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PRCP0600
PRCP0610
PRCP0620
PRCP0630
PRCP0640
PRCP0650
PRCP0660
PRCP0670
PRCP0680
PRCP0690
PRCP0700
PRCP0710
PRCP0720
PRCP0730
PRCP0740
PRCP0750
PRCP0760
PRCP0770
PRCP0780
PRCP0790
PRCP0800
PRCP0810
PRCP0820
PRCP0830
PRCP0840
PRCP0850
PRCP0860
PRCP0870
PRCP0880
PRCP0890
PRCP0900
PRCP0910
PRCP0920
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PRCP0940
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PRCP0960
PRCP0970
PRCP0980
PRCP0990
PRCP1000
PRCP1010
PRCP1020
PRCP1030
PRCP1040
PRCP1050
PRCP1060
PRCP1070
PRCP1080
PRCP1090
PRCP1100
PRCP1110
PRCP1120
PRCP1130
PRCP1140
PRCP1150
PRCP1160
PRCP1170
PRCP1180
PRCP1190

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C      VISLP = ((1.0167D-03)*(T**1.51))/(T+5.8655D+02)
      XIN = ((4.18878D+02)**0.16667)
      XID = (1.70936D+02**0.5)*(3.2189D+01**0.56667)
      XI = (XIN)/(XID)
      DENR = (DENSTY)/(3.632D+01)
      XIC = (2.5D-04)/(XI)
      VISHP = (VISLP)+(1.2312D-04)*(DEXP(1.079D+00*DENR))/(XI) - (XIC)
      WISC(J)=VISHP
      PRCP120C
      PRCP121C
      PRCP1220
      PRCP1230
      PRCP124C
      PRCP1250
      PRCP1260
      PRCP127C
      PRCP128C
      PRCP1290
      PRCP1300
      PRCP1310
      PRCP1320
      PRCP1330
      PRCP1340
      PRCP1350
      PRCP1360
      PRCP137C
      PRCP1380
      PRCP1390
      PRCP1400
      PRCP141C
      PRCP1420
      PRCP1430
      PRCP1440
      PRCP1450
      PRCP1460
      PRCP1470
      PRCP1480
      PRCP1490
      PRCP1500
      PRCP1510
      PRCP1520
      PRCP1530
      PRCP1540
      PRCP1550
      PRCP1560
      PRCP1570
      PRCP158C
      PRCP1590
      PRCP1600
      PRCP1610
      PRCP1620
      PRCP1630
      PRCP1640
      PRCP1650
      PRCP1660
      PRCP1670
      PRCP1680
      PRCP1690
      PRCP1700
      PRCP1710
      PRCP1720
      PRCP1730
      PRCP1740
      PRCP1750
      PRCP1760
      PRCP1770
      PRCP1780
      PRCP1790

50 ITR=1
C      RETURN
      END
      PRCP1800
      PRCP1810
      PRCP1820
      PRCP1830

C      COMPUTING THE DERIVATIVES OF THE ABSOLUTE VISCOSITY EQUATION.
      DERUDT = (1.079D+00/3.632D+01)*(VISHP-VISLP+XIC)
      DERUDT = (1.5)*(VISLP/T)-(VISLP)/(T+5.8655D+02)
      DERUTP = (DERUDT)*(DERDTP)+(DERUDT)
      PRCP1300
      PRCP1310
      PRCP1320
      PRCP1330
      PRCP1340
      PRCP1350
      PRCP1360
      PRCP137C
      PRCP1380
      PRCP1390
      PRCP1400
      PRCP141C
      PRCP1420
      PRCP1430
      PRCP1440
      PRCP1450
      PRCP1460
      PRCP1470
      PRCP1480
      PRCP1490
      PRCP1500
      PRCP1510
      PRCP1520
      PRCP1530
      PRCP1540
      PRCP1550
      PRCP1560
      PRCP1570
      PRCP158C
      PRCP1590
      PRCP1600
      PRCP1610
      PRCP1620
      PRCP1630
      PRCP1640
      PRCP1650
      PRCP1660
      PRCP1670
      PRCP1680
      PRCP1690
      PRCP1700
      PRCP1710
      PRCP1720
      PRCP1730
      PRCP1740
      PRCP1750
      PRCP1760
      PRCP1770
      PRCP1780
      PRCP1790

C      COMPUTING THE THERMAL CONDUCTIVITY (CAL/SEC CM DEGK).
      CONC = ((1.32D-02)*(2.52D+02)*(1.8D+03))/(4.536D+02)
      CON1 = (CONC)*(CVLP)*(VISLP)
      CON2 = ((3.40D-02)*(VISLP))/(1.70936D+02)
      CON3 = ((7.0D-03)*(VISLP))/(TR)*(1.70936D+02)
      CONLP = (CON1)+(CON2)-(CON3)
      CON5 = (4.73187D+02)*(1.70936D+02)
      CON6 = (3.632D+01)*(1.07315D+01)*(7.5395D+02)
      CON7C = (CON5)/(CON6)
      CONL1 = (1.70936D+02**0.51)*(4.18877D+02**0.16667)
      CONL2 = (3.2189D+01**0.66667)
      CONLA = CONL1/CONL2
      CON7 = (CONLA)*(CON7C**5.0)
      CONHP = (CONLP)+(1.31D-07*DEXP(6.7D-01*DENR))/CON7)
      I = (1.40039D-07/CON7)
      COND(J)=CONHP
      PRCP1510
      PRCP1520
      PRCP1530
      PRCP1540
      PRCP1550
      PRCP1560
      PRCP1570
      PRCP158C
      PRCP1590
      PRCP1600
      PRCP1610
      PRCP1620
      PRCP1630
      PRCP1640
      PRCP1650
      PRCP1660
      PRCP1670
      PRCP1680
      PRCP1690
      PRCP1700
      PRCP1710
      PRCP1720
      PRCP1730
      PRCP1740
      PRCP1750
      PRCP1760
      PRCP1770
      PRCP1780
      PRCP1790

C      COMPUTING THE DERIVATIVES OF THE THERMAL CONDUCTIVITY EQUATION.
      DERKDT = (6.7D-01/3.632D+01)*(CONHP-CONLP+(1.40039D-07/CON7))
      DERCVT = (3.49D-04) - (3.34D-07*T)
      DER1 = (CONC)*(CVLP)*(DERUDT)
      DER2 = (CONC)*(VISLP)*(DERCVT)
      DER3 = (3.40D-02/1.70936D+02)*(DERUDT)
      DER4 = ((7.0D-03)*(DERUDT))/((1.70936D+02)*(TR))
      DER5 = ((7.0D-03)*(VISLP))/(1.70936D+02)*(TR)*(T)
      DERKTD = (DER1)+(DER2)+(DER3)-(DER4)+(DER5)
      DERKTP = (DERKDT)*(DERDTP) + (DERKTD)
      PRCP1510
      PRCP1520
      PRCP1530
      PRCP1540
      PRCP1550
      PRCP1560
      PRCP1570
      PRCP158C
      PRCP1590
      PRCP1600
      PRCP1610
      PRCP1620
      PRCP1630
      PRCP1640
      PRCP1650
      PRCP1660
      PRCP1670
      PRCP1680
      PRCP1690
      PRCP1700
      PRCP1710
      PRCP1720
      PRCP1730
      PRCP1740
      PRCP1750
      PRCP1760
      PRCP1770
      PRCP1780
      PRCP1790

C      COMPUTING THE COEFF'S OF THE DIFF. EQUATIONS.
      DCOEFF = (DERDTP)/(DENSTY)
      ECOEFF = (DERUTP)/(VISHP)
      HCOEFF = (DERKTP)/(CONHP)
      PRNC = (CPHP*VISHP+1.0D-02)/(CONHP)
      PRCP1670
      PRCP1680
      PRCP1690
      PRCP1700
      PRCP1710
      PRCP1720
      PRCP1730
      PRCP1740
      PRCP1750
      PRCP1760
      PRCP1770
      PRCP1780
      PRCP1790

C      AA1(J)=DCOEFF + ECOEFF
      AA2(J)=1.0D+00/(DENSTY*VISHP)
      AA3(J)=1.0D+00/(DENSTY*DENSTY*VISHP)
      BB1(J)=DCOEFF + HCOEFF
      BB2(J)=(CPHP)/(DENSTY*CONHP)
      PRCP1670
      PRCP1680
      PRCP1690
      PRCP1700
      PRCP1710
      PRCP1720
      PRCP1730
      PRCP1740
      PRCP1750
      PRCP1760
      PRCP1770
      PRCP1780
      PRCP1790

C      T=T+DTEMP

```

```

SUBROUTINE STEPT (FUNK)
C
C INTERFACE TO MAKE MARQ LOOK LIKE STEPT.
C
C TO USE THIS ROUTINE, SET THE VALUE OF LPCOL AND THE DIMENSIONS OF
C THE ARRAYS P, FITSV, FIT, Y, AND YSIG. THE DIMENSIONS ARE....
C P(LPCOL,NVMAX), FITSV(LPCOL), FIT(LPCOL), Y(LPCOL), YSIG(LPCOL)
C WHERE LPCOL IS THE MAXIMUM VALUE OF NPTS AND NVMAX IS THE MAXIMUM
C VALUE OF NV. (IF LEQU.EQ.1, YSIG MAY BE DIMENSIONED YSIG(1).)
C COMMON/CDAT/ DOES NOT APPEAR IN ANY ROUTINE OF THE MARQ PACKAGE
C OTHER THAN THIS ONE, SO THAT THIS IS THE ONLY ONE WHICH MUST BE
C RECOMPILED WHEN LPCOL AND THE DIMENSIONS OF THE ARRAYS ARE CHANGED.
C
C THE FOLLOWING EXTERNAL STATEMENT IS REQUIRED BY SOME COMPILERS
C (MATFIV, FOR EXAMPLE) AND FORBIDDEN BY OTHERS (MODCOMP II).
C EXTERNAL FUNK
C
C DOUBLE PRECISION FIT,FITSV
C
C DIMENSION P(275,8),FITSV(275)
C DIMENSION P(25,20),FITSV(25)
C
C COMMON /CDAT/ FIT(275),Y(275),YSIG(275),NPTS
C COMMON /CDAT/ FIT(25),Y(25),YSIG(25),NPTS
C
C LPCOL=1
C LPCOL=275
C LPCOL=25
C
C CALL MARQ (FUNK,Y,YSIG,NPTS,FIT,FITSV,P,LPCOL)
C
C RETURN
C END
C
C SUBROUTINE STSET
C
C STSET SETS SOME INPUT QUANTITIES TO DEFAULT VALUES, FOR MARQ.
C
C NOTE.....
C THIS VERSION OF STSET MAY ALSO BE USED WITH STEFIT, SIMPLEX,
C STP, KAUPE, AND MINF.
C
C USAGE.....
C CALL STSET. THEN SET SOME INPUT QUANTITIES (NV AND NPTS, AT LEAST)
C AND RESET ANY OF THOSE SET IN STSET (BETTER VALUES OF X, ETC.)
C BEFORE CALLING MARQ OR THE STEPT-MARQ INTERFACE ROUTINE.
C
C DOUBLE PRECISION X
C
C COMMON /CSTEP/ X(20),XMAX(20),XMIN(20),DELTX(20),DELMN(20),
C * ERFI(20,21),FDBJ,NV,NTRAC,MATRIX,MASK(20),
C * NFMAX,NFLAT,JVARY,NXTRA,KFLAG,NOREP,KERFL,KW
C COMMON /NLS2/FLAMB,RELOF,METHD,KALCP,KORDF,MAXIT,LEQU,NXSUB,MXUPD
C
C KW ... LOGICAL UNIT NUMBER OF THE PRINTER
C
C KW=6
C NVMAX=20
C
C THE USER MUST SET NV AFTER CALLING STSET.

```

```

STMARQ 1
STMARQ 2
STMARQ 3
STMARQ 4
STMARQ 5
STMARQ 6
STMARQ 7
STMARQ 8
STMARQ 9
STMARQ10
STMARQ11
STMARQ12
STMARQ13
STMARQ14
STMARQ15
STMARQ16
STMARQ17
STMARQ18
STMARQ19
STMARQ20
STMARQ21
STMARQ22
STMARQ23
STMARQ24
STMARQ25
STMARQ26
STMARQ27
STMARQ28
STMARQ29
STMARQ30
STMARQ31
STMARQ32
STMARQ33
STMARQ35
STMARQ36
STSETP 1
STSETP 2
STSETP 3
STSETP 4
STSETP 5
STSETP 6
STSETP 7
STSETP 8
STSETP 9
1000TM 9
2000TM 9
3000TM 9
STSETP13
STSETP14
STSETP15
STSETP16
STSETP17
STSETP18
STSETP19
STSETP20
STSETP21
STSETP22
STSETP23
STSETP24

```

```

NV=-1
NPTS=-1
HUGE=1.E30
RZERO=0.
NTRAC=0
NFMAX=22767
MAXIT=20
NXSUB=50
METHD=1
KALCP=0
KORDF=1
LEQU=0
NFLAT=1
MATRIX=106
NXTRA=0
FLAMB=1.
RELOF=1.E-8
C
C JC 10 JX=1,NVMAX
C A(JX)=RZERO
C XMAX(JX)=HUGE
C XMIN(JX)=-HUGE
C DELTX(JX)=RZERO
C DELMNI(JX)=RZERO
C MASK(JX)=0
10 CONTINUE
C
C RETURN
C END
C
C SUBROUTINE FOFX (JPT,NV,X,F)
C
C THIS IS A DUMMY VERSION OF SUBROUTINE FOFX.
C A NON-DUMMY VERSION OF FOFX MAY BE USED (OPTIONALLY) TO SUPPLY
C TO MARQ VALUES OF THE FUNCTION BEING FITTED, INSTEAD OF USING A
C -FUNK- SUBROUTINE TO DO THIS. THE USE OF FOFX REQUIRES
C SUBSTANTIALLY MORE OVERHEAD TIME DURING EXECUTION, BUT SAVES
C CONSIDERABLE STORAGE BY NOT REQUIRING THAT THE JACOBIAN MATRIX, P,
C BE STORED. THE MINI-MARQ PACKAGE REQUIRES THE USE OF FOFX.
C TO SEE HOW TO USE A NON-DUMMY FOFX, SEE THE LISTING OF MARQ.
C
C DOUBLE PRECISION X,F
C DIMENSION X(20)
C RETURN
C END
C
C SUBROUTINE CALCD (JPT,P,LPCOL)
C
C THIS IS A DUMMY VERSION OF SUBROUTINE CALCD.
C A NON-DUMMY VERSION OF CALCD MAY BE USED (OPTIONALLY) TO SUPPLY
C TO MARQ OR MINI-MARQ ANALYTIC VALUES OF THE ELEMENTS OF THE
C JACOBIAN MATRIX, INSTEAD OF CALCULATING THEM USING FINITE
C DIFFERENCES. MOST USERS USE FINITE DIFFERENCES MOST OF THE TIME.
C TO SEE HOW TO USE A NON-DUMMY CALCD, SEE THE LISTING OF MARQ.
C
C DIMENSION P(LPCOL,1)
C RETURN
C END
C
C SUBROUTINE MARQ (FUNK,Y,YSIG,NPTS,FIT,FITSV,P,LPCOL)

```

```

STSETM25
STSETM26
STSETM27
STSETM28
STSETM29
STSETM30
STSETM31
STSETM32
STSETM33
STSETM34
STSETM35
STSETM36
STSETM37
STSETM38
STSETM39
STSETM40
STSETM41
STSETM42
STSETM43
STSETM44
STSETM45
STSETM46
STSETM47
STSETM48
STSETM49
STSETM50
STSETM51
STSETM52
STSETM54
STSETM55
FOFXDUM1
FOFXDUM2
FOFXDUM3
FOFXDUM4
FOFXDUM5
FOFXDUM6
FOFXDUM7
FOFXDUM8
FOFXDUM9
FOFXDUM10
FOFXDUM11
1000DUM11
2000DUM11
FOFXDUM12
FOFXDUM13
FOFXDUM14
CALCDUM1
CALCDUM2
CALCDUM3
CALCDUM4
CALCDUM5
CALCDUM6
CALCDUM7
CALCDUM8
CALCDUM9
1000DUM9
CALCDUM10
CALCDUM11
CALCDUM12
MARQ 1

```



```

C
C MARQ 2.5      A.N.S.I. STANDARD FORTRAN      AUGUST 1976      MARQ 2
C COPYRIGHT (C) 1978 J. P. CHANDLER      MARQ 3
C
C J. P. CHANDLER AND LEON W. JACKSON,      MARQ 4
C DEPARTMENT OF COMPUTING AND INFORMATION SCIENCES      MARQ 5
C OKLAHOMA STATE UNIVERSITY, STILLWATER, OKLAHOMA 74074      MARQ 6
C
C MARQ PERFORMS A NONLINEAR LEAST SQUARES FIT OF A USER-SUPPLIED      MARQ 7
C FUNCTION TO A GIVEN SET OF DATA, USING MARQUARDT-S METHOD, OR THE      MARQ 8
C GAUSS-NEWTON METHOD, OR A MODIFIED GAUSS-NEWTON METHOD.      MARQ 9
C D. W. MARQUARDT, J.SOC.IND.APPL.MATH. 11 (1963) 431-441      MARQ 10
C
C *****      MARQ 11
C
C INPUT QUANTITIES..... FUNK,X(I*),XMIN(I*),MASK(I*),DELMN(I*),MATRIX,      MARQ 12
C Y(I*),YSIG(I*),NV,NPTS,LPCOL,FLAMB,NFMAX,      MARQ 13
C NTRAC,KALCP,KORDF,PETHD,NFLAT,RELD,      MARQ 14
C MAXIT,LEQU,KW      MARQ 15
C
C OUTPUT QUANTITIES.... X(I*),FDBJ,ERR(I*),KFLAG,FIT(I*)      MARQ 16
C UNUSED QUANTITIES (INCLUDED FOR COMPATIBILITY WITH STEPT COMMON)....      MARQ 17
C DELTA(I*),JVVARY,NXTRA,NOREP,KERFL      MARQ 18
C
C FUNK      -- THE NAME OF THE FUNCTION CALLED TO OBTAIN      MARQ 19
C THE FITTED VALUES IF KALCP=0      MARQ 20
C NV      -- THE NUMBER OF PARAMETERS      MARQ 21
C X(IJX)      -- THE JX-TH PARAMETER      MARQ 22
C FDBJ      -- RETURNS THE FINAL VALUE OF PHI, THE WEIGHTED      MARQ 23
C SUM OF SQUARES (PHI IS BEING MINIMIZED AS A      MARQ 24
C FUNCTION OF THE X(IJX))      MARQ 25
C XMAX(IJX)      -- THE UPPER LIMIT FOR X(IJX)      MARQ 26
C XMIN(IJX)      -- THE LOWER LIMIT FOR X(IJX)      MARQ 27
C MASK(IJX)      -- NONZERO IF X(IJX) IS TO BE HELD FIXED      MARQ 28
C DELMNIJX      -- THE CONVERGENCE TOLERANCE FOR X(IJX)      MARQ 29
C NTRAC      -- USER PRINT CONTROL      MARQ 30
C      =-3, NO OUTPUT EXCEPT FATAL ERROR MESSAGES      MARQ 31
C      =-2, NO OUTPUT EXCEPT DIAGNOSTIC MESSAGES      MARQ 32
C      =-1, STANDARD OUTPUT EXCEPT FINAL FIT VALUES      MARQ 33
C      = 0, STANDARD OUTPUT      MARQ 34
C      = 1, ALSO PRINTS RESULTS OF EACH ITERATION      MARQ 35
C      = 2, ALSO PRINTS THE COEFFICIENT MATRIX, QSAV      MARQ 36
C      = 3, ALSO PRINTS THE JACOBIAN MATRIX, P      MARQ 37
C NFMAX      -- THE MAXIMUM NUMBER OF FUNCTION COMPUTATIONS      MARQ 38
C NFLAT      -- NONZERO IF THE SEARCH IS TO TERMINATE WHEN TWO      MARQ 39
C ITERATIONS GIVE IDENTICAL VALUES OF PHI      MARQ 40
C KW      -- THE LOGICAL UNIT NUMBER OF THE PRINTER      MARQ 41
C ERR(IJX,KX)      -- RETURNS THE ERROR MATRIX      MARQ 42
C KFLAG      -- RETURNED .GT. ZERO FOR A NORMAL EXIT,      MARQ 43
C RETURNED .LT. ZERO FOR AN ABNORMAL EXIT      MARQ 44
C
C NPTS      -- THE NUMBER OF DATA OBSERVATIONS      MARQ 45
C Y(JPT)      -- THE JPT-TH DATA ORDINATE      MARQ 46
C YSIG(JPT)      -- THE EXPECTED ERROR IN Y(JPT)      MARQ 47
C FIT(JPT)      -- THE JPT-TH FITTED VALUE      MARQ 48
C FITSV      -- A SCRATCH VECTOR OF NPTS VALUES      MARQ 49
C P(IJPT,JX)      -- THE FIRST PARTIAL DERIVATIVE      MARQ 50
C OF FIT(JPT) WITH RESPECT TO X(IJX)      MARQ 51
C LPCOL      -- THE FIRST DIMENSION OF THE ARRAY CONTAINING P      MARQ 52
C (LPCOL MUST BE .GE. NPTS IF KALCP IS .GE. ZERO)      MARQ 53
C LEQU      -- SAVES STORAGE IF ALL YSIG(JPT) ARE THE SAME      MARQ 54

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C      =1 IF ALL YSIG(JPT) ARE EQUAL (IN THIS CASE      MARQ 55
C ONLY YSIG(1) IS REFERENCED)      MARQ 56
C      =0 OTHERWISE      MARQ 57
C FLAMB      -- MARQUARDT-S LAMBDA, THE RELATIVE AMOUNT BY WHICH      MARQ 58
C THE DIAGONAL COEFFICIENTS OF THE NORMAL      MARQ 59
C EQUATIONS ARE AUGMENTED      MARQ 60
C MXSUB      -- THE MAXIMUM NUMBER OF SUBITERATIONS PERMITTED      MARQ 61
C KALCP      -- DETERMINES WHICH ROUTINE IS CALLED TO COMPUTE THE      MARQ 62
C JACOBIAN MATRIX OF FIRST PARTIAL DERIVATIVES      MARQ 63
C      =-1, ONE ROW OF P RETURNED BY DERIV OR CALCO,      MARQ 64
C      DERIV CALLS FOFX      MARQ 65
C      = 0, ALL OF P RETURNED BY DERIV OR CALCO,      MARQ 66
C      DERIV CALLS FUNK      MARQ 67
C      = 1, ALL OF P RETURNED BY DERIV OR CALCO,      MARQ 68
C      DERIV CALLS FOFX      MARQ 69
C KORDF      -- DETERMINES HOW P IS TO BE CALCULATED      MARQ 70
C      = 1, FIRST ORDER APPROXIMATION USED BY DERIV      MARQ 71
C      = 2, SECOND ORDER APPROXIMATION USED BY DERIV      MARQ 72
C      = 3, ANALYTICAL DERIVATIVE SUPPLIED BY CALCO      MARQ 73
C METHD      -- DETERMINES THE METHOD USED      MARQ 74
C      =-1, MODIFIED GAUSS-NEWTON METHOD      MARQ 75
C      = 0, GAUSS-NEWTON METHOD      MARQ 76
C      = 1, MODIFIED FORM OF MARQUARDT-S METHOD      MARQ 77
C      = 2, MARQUARDT-S METHOD      MARQ 78
C RELD      -- DETERMINES THE MAGNITUDE OF THE DIFFERENCING STEP      MARQ 79
C MAXIT      -- THE MAXIMUM NUMBER OF ITERATIONS TO BE PERFORMED      MARQ 80
C
C *MODIFICATIONS TO MARQUARDT-S METHOD...      MARQ 81
C 1. THE QUANTITY (1+LAMBDA) IS NOT ALLOWED TO INCREASE BY MORE      MARQ 82
C THAN A FACTOR OF TWO, IN ANY SINGLE INCREASE.      MARQ 83
C 2. WHEN CUTSTEPS ARE USED IN A LINE SEARCH (WHEN THE ANGLE      MARQ 84
C GAMMA .LT. GAMMA SUB ZERO), THE VALUE OF LAMBDA IS INCREASED      MARQ 85
C PROPORTIONATELY.      MARQ 86
C IN BOTH THE MODIFIED MARQUARDT-S METHOD AND THE MODIFIED      MARQ 87
C GAUSS-NEWTON METHOD, A HALF STEP IS ATTEMPTED FOLLOWING EACH      MARQ 88
C SUCCESSFUL STEP. THIS AVOIDS ONE FORM OF VERY SLOW CONVERGENCE.      MARQ 89
C
C *****      MARQ 90
C THE FOLLOWING EXTERNAL STATEMENT IS REQUIRED BY SOME COMPILERS      MARQ 91
C (WATFIV, FOR EXAMPLE) AND FORBIDDEN BY OTHERS (MDDCOMP II).      MARQ 92
C EXTERNAL FUNK      MARQ 93
C
C ON A MACHINE HAVING LESS THAN ABOUT TEN SIGNIFICANT DIGITS IN      MARQ 94
C SINGLE PRECISION (FOR EXAMPLE THE IBM 360 OR 370), IF P IS      MARQ 95
C COMPUTED USING FINITE DIFFERENCES (KORDF.LT.3), THIS PART OF THE      MARQ 96
C COMPUTATION SHOULD BE DONE IN DOUBLE PRECISION. TO ACCOMPLISH      MARQ 97
C THIS, ACTIVATE THE DOUBLE PRECISION STATEMENT IN EACH SUBROUTINE.      MARQ 98
C ORDINARILY THE OTHER COMPUTATIONS MAY BE DONE IN SINGLE PRECISION.      MARQ 99
C
C DOUBLE PRECISION X,XSAVE,XTEMP,GRAD,FIT,FITSV,      MARQ 100
C * XSAV,SIG,RTERM,YY,PHI,PHNE,PHALF,XLIN,STFAC      MARQ 101
C
C THE DIMENSIONS OF THE VECTORS AND MATRICES (AS OPPOSED TO ARRAYS)      MARQ 102
C ARE....      MARQ 103
C P(NPTS,NACTV) (OR P(1,NACTV) IF KALCP.EQ.-1),      MARQ 104
C FITSV(NPTS) (OR FITSV(1) IF KALCP.EQ.-1),      MARQ 105
C X(INV),XMAX(INV),XMIN(INV),DELMN(INV),ERR(INV,NV+1),      MARQ 106
C XSAVE(INV),H(INV),MASK(INV),      MARQ 107
C GRAD(INACTV),SCALE(INACTV),      MARQ 108

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C      Y(NPTS),FIT(NPTS),YSIG(NPTS) (OR YSIG(1) IF LEQU,NE,0),      MARQ 122
C      WHERE NACTV IS THE NUMBER OF ACTIVE (UNMASKED) X(J).      MARQ 123
C                                                                MARQ 124
C      DIMENSION P(LPCOL,1)      MARQ 125
C      DIMENSION Y(1),YSIG(1),FIT(1),FITSV(1)      MARQ 126
C      DIMENSION XSAVE(20),H(2C),GRAD(2C),SCALE(2C),MASKT(2C),XTEMP(2C)      MARQ 127
C                                                                MARQ 128
C USER COMMON.....      MARQ 129
C COMMON /CSTEP/ X(20),XMAX(20),XMIN(20),DELTX(20),DELMN(20),      MARQ 130
C *   ERR(20,21),FDBJ,NV,NTRAC,MATRX,MASK(20),      MARQ 131
C *   NFMAX,NFLAT,JVARY,NXTRA,KFLAG,NOREP,KERFL,KW      MARQ 132
C                                                                MARQ 133
C MARQ COMMON.....      MARQ 134
C COMMON /NLS2/FLAMB,RELD,FMETHD,KALCP,KORDF,MAXIT,LEQU,MXSUB,MXUPD      MARQ 135
C                                                                MARQ 136
C SET THE LIBRARY FUNCTION FOR SINGLE PRECISION (SORT) OR FOR      MARQ 137
C DOUBLE PRECISION (DSORT). NO OTHER FUNCTIONS ARE USED, EITHER      MARQ 138
C EXTERNAL OR INTRINSIC.      MARQ 139
C THE ONLY SUBROUTINES CALLED ARE FUNC, DERIV, CALCD, AND RWERR.      MARQ 140
C                                                                MARQ 141
C      QSQR(TARG)=SQRT(TARG)      MARQ 142
C                                                                MARQ 143
C * * * * *      MARQ 144
C                                                                MARQ 145
C SET FIXED QUANTITIES ....      MARQ 146
C                                                                MARQ 147
C                                                                MARQ 148
C                                                                MARQ 149
C                                                                MARQ 150
C                                                                MARQ 151
C                                                                MARQ 152
C NVMAX IS THE MAXIMUM PERMISSIBLE VALUE OF NV. IT IS ALSO THE      MARQ 153
C DIMENSION OF THE ARRAYS X, XMAX, XMIN, MASK, DELMN,      MARQ 154
C XSAVE, H, MASKT, GRAD, AND SCALE, AND THE FIRST DIMENSION OF ERR.      MARQ 155
C THE SECOND DIMENSION OF ERR IS NVMAX+1.      MARQ 156
C                                                                MARQ 157
C NVMAX=20      MARQ 158
C                                                                MARQ 159
C FNU ... MARQUARDT-S NU, THE FACTOR BY      MARQ 160
C WHICH FLAMB IS CHANGED      MARQ 161
C                                                                MARQ 162
C FNU=10.      MARQ 163
C                                                                MARQ 164
C CRIT ... COSINE OF MARQUARDT-S CRITICAL      MARQ 165
C ANGLE, GAMMA SUB ZERO      MARQ 166
C                                                                MARQ 167
C CRIT=.70711      MARQ 168
C                                                                MARQ 169
C FLDEF ... DEFAULT VALUE FOR FLAMB      MARQ 170
C                                                                MARQ 171
C FLDEF=1.      MARQ 172
C                                                                MARQ 173
C RELMN ... USED TO SET DEFAULT VALUE OF      MARQ 174
C DELMN      MARQ 175
C                                                                MARQ 176
C RELMN=1.E-5      MARQ 177
C                                                                MARQ 178
C RLTL ... TOLERANCE FOR A WARNING MESSAGE      MARQ 179
C                                                                MARQ 180
C RLTL=1.E-5      MARQ 181
C                                                                MARQ 182
C RZERO=0.      MARQ 183
C RUNIT=1.      MARQ 184
C RTWC=2.      MARQ 185
C                                                                MARQ 186
C NO REAL CONSTANTS ARE USED BEYOND THIS POINT.      MARQ 187
C                                                                MARQ 188
C NTRPT=NTRAC-4-2)      MARQ 189
C NTRPU=NTRAC-(1)      MARQ 190
C NTRMU=NTRAC-1      MARQ 191
C KFLAG=0      MARQ 192
C                                                                MARQ 193
C                                                                MARQ 194
C                                                                MARQ 195
C                                                                MARQ 196
C                                                                MARQ 197
C                                                                MARQ 198
C                                                                MARQ 199
C                                                                MARQ 200
C                                                                MARQ 201
C                                                                MARQ 202
C                                                                MARQ 203
C                                                                MARQ 204
C                                                                MARQ 205
C                                                                MARQ 206
C                                                                MARQ 207
C                                                                MARQ 208
C                                                                MARQ 209
C                                                                MARQ 210
C                                                                MARQ 211
C                                                                MARQ 212
C                                                                MARQ 213
C                                                                MARQ 214
C                                                                MARQ 215
C SET X(JX) TO AMAX1(XMIN(JX),AMIN1(XMAX(JX),X(JX))).      MARQ 216
C                                                                MARQ 217
C                                                                MARQ 218
C                                                                MARQ 219
C                                                                MARQ 220
C                                                                MARQ 221
C                                                                MARQ 222
C                                                                MARQ 223
C                                                                MARQ 224
C                                                                MARQ 225
C                                                                MARQ 226
C                                                                MARQ 227
C                                                                MARQ 228
C                                                                MARQ 229
C                                                                MARQ 230
C                                                                MARQ 231
C                                                                MARQ 232
C                                                                MARQ 233
C                                                                MARQ 234
C                                                                MARQ 235
C                                                                MARQ 236
C                                                                MARQ 237
C                                                                MARQ 238
C                                                                MARQ 239
C                                                                MARQ 240
C                                                                MARQ 241
C                                                                MARQ 242

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```

1290 FORMAT(/10H DELTA = ,8E13.5/(10X,8E13.5))
      WRITE(KW,1300)NY,NPTS,LPCOL,ATRAC,METHD,KALCP,KORDF,NFLAT,
      * NFMX,MAXIT,MXSUB,CRIT
1300 FORMAT(/6H NV = ,I4,5X,8H NPTS = ,I6,5X,9H LPCOL = ,I6,
      * 5X,9H NTRAC = ,I2,5X,9H METHD = ,I2,5X,9H KALCP = ,I2,
      * 5X,9H KORDF = ,I2//9H NFLAT = ,I2,5X,9H NFMX = ,I7,
      * 5X,9H MAXIT = ,I5,5X,9H MXSUB = ,I5,5X,
      * 28H COSINE OF CRITICAL ANGLE = ,E12.5)
1310 JVVARY=0
C
C SET FMGN, FLUPB, AND, IF NECESSARY, FLAMB.
C
      FMGN=RUNIT
      FLUPB=RTWO
      IF(METHD)1320,1320,1330
1320 FLAMB=RZERO
      GO TO 1390
1330 IF(FLAMB)1340,1340,1350
1340 FLAMB=FLDEF
1350 IF(METHD-2)1390,1380,1390
1390 FLUPB=RUNIT
C
C COMPUTE THE INITIAL GOODNESS OF FIT OF THE MODEL TO THE DATA.
C CALL FUNC TO CALCULATE THE VECTOR OF FITTED VALUES.
C
1390 CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHI)
C
C
      NF ... EQUIVALENT NUMBER OF CALLS TO FUNC
      NF=1
      IF(NTRPU)1420,1400,1400
1400 WRITE(KW,1410)PHI,FLAMB
1410 FORMAT(/28H PHI (THE SUM OF SQUARES) = ,E15.8,56X,9H LAMBDA = ,
      * E12.5//1H )
C
C *****
C
C BEGIN THE NEXT ITERATION.
C THIS IS THE ENTRY POINT AFTER A SUCCESSFUL STEP IF THE CONVERGENCE
C CRITERION IS NOT MET.
C
1420 JSUB=0
      ITER=ITER+1
      IF(NTRMU)1425,1421,1421
1421 WRITE(KW,1422)ITER,FMGN,FLAMB
1422 FORMAT(/17H BEGIN ITERATION ,I4,4X,7H FMGN =,E12.5,15X,
      * 9H LAMBDA =,E12.5)
1425 IF(NTRAC-3)1450,1430,1430
1430 WRITE(KW,1440)
1440 FORMAT(/28H P (THE JACOBIAN MATRIX)..../1H )
C
C INITIALIZE FOR THIS ITERATION.
1450 STFAC=RUNIT
      DO 1460 JX=1,NACTV
      GRAD(JX)=RZERO
      DO 1460 KX=1,NACTV
1460      ERR(JX,KX)=RZERO
C
C CALL DERIV (OR CALCD) TO COMPUTE THE JACOBIAN MATRIX, P.
C DERIV IS CALLED NPTS TIMES IF KALCP=-1.
C

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SIG=YSIG(1)
DC 1590 JPT=1,NPTS
      KPT=JPT
      IF(KALCP)1470,1490,1490
1470      KPT=1
      IF(KORDF-2)1500,1500,1480
1490      CALL CALCD (JPT,P,LPCOL)
      GO TO 1510
1490      IF(JPT-1)1510,1500,1510
1500      CALL DERIV (JPT,FUNK,NPTS,FIT,FITSV,P,LPCOL)
1510      CONTINUE
      IF(NTRAC-3)1540,1520,1520
1520      WRITE(KW,1530)JPT,(P(KPT,JX),JX=1,NACTV)
1530      FORMAT(1X,I3,2X,6E15.7/(6X,6E15.7))
C
C COMPUTE QSAV AND GRAD.
C QSAV, WHICH IS STORED IN ONE HALF OF THE ARRAY ERR(***), IS PT*P.
C GRAD IS EQUAL TO HALF THE NEGATIVE OF THE GRADIENT VECTOR.
C
1540      IF(LEQU)1560,1550,1560
1550      SIG=YSIG(JPT)
1560      YY=Y(JPT)
      RTERM=(FIT(JPT)-YY)/SIG**2
      DO 1580 JX=1,NACTV
      GRAD(JX)=GRAD(JX)-P(KPT,JX)*RTERM
      PTERM=P(KPT,JX)/SIG**2
      DO 1570 KX=1,JX
      ERR(JX,KX)=ERR(JX,KX)+P(KPT,KX)*PTERM
      CONTINUE
1590      CONTINUE
C
      RESTORE FIT IF IT WAS DESTROYED IN DERIV.
1600      IF(KORDF-2)1630,1600,1630
1610      IF(KALCP)1620,1610,1620
1610      CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHI)
      NF=NF+1
1620      NF=NF+NACTV
1630      NF=NF+NACTV
C
C COMPUTE THE SCALE FACTORS AND SCALE GRAD.
C
      DO 1660 JX=1,NACTV
      SCALJ=OSQRT(ERR(JX,JX))
      IF(SCALJ)1650,1640,1650
1640      SCALJ=RUNIT
1650      SCALE(JX)=SCALJ
1660      GRAD(JX)=GRAD(JX)/SCALJ
      IF(NTRMU)1690,1670,1670
1670      WRITE(KW,1680)(GRAD(JX),JX=1,NACTV)
1680      FORMAT(/2X,19H SCALED GRADIENT = ,6E15.7/(21X,6E15.7))
C
C SCALE QSAV. THE DIAGONAL ELEMENTS OF QSAV ARE SCALED TO UNITY.
C
1690      DO 1770 JX=1,NACTV
      DO 1770 KX=1,JX
      SA=ERR(JX,KX)/(SCALE(JX)*SCALE(KX))
      IF(KX-JX)1720,1700,1720
1700      IF(SA)1710,1740,1710
1710      SA=RUNIT
      GO TO 1770
1720      IF(SA-(RUNIT-RLTOL))1730,1740,1740

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1730 IF(SA+(RUNIT-RLTOL))1740,1740,1770      PARQ 362
1740 CONTINUE                                PARQ 363
      IF(INTRPT)1770,1750,1750              PARQ 364
1750 WRITE(KW,1760)JX,KX,SA                 PARQ 365
1760 FORMAT(38H ***** POSSIBLY DANGEROUS VALUE OF ,
      * 16H COEFFICIENT.....5X,6H QSAV(I,I3,1H,;I3,4H) = ,E14.7)
1770 ERR(JX,KX)=SA                           PARQ 366
      IF(INTRAC-2)1810,1780,1780            PARQ 369
1780 WRITE(KW,1790)                          PARQ 370
1790 FORPAT(/49H QSAV (PT=P, SCALED, WHERE P IS THE JACOBIAN)...../1H )
      DO 1800 JX=1,NACTV                    PARQ 371
1800 WRITE(KW,1530)JX,(ERR(JX,KX),KX=1,JX)  PARQ 372
1810 DO 1820 JX=1,NV                         PARQ 373
1820 XSAVE(JX)=X(JX)                        PARQ 374
C                                             PARQ 375
C          INITIALIZE MASKT AND NACT.        PARQ 376
1830 NACT=NACTV                              PARQ 377
      DO 1840 JX=1,NV                       PARQ 378
1840 MASKT(JX)=MASK(JX)                    PARQ 379
C                                             PARQ 380
C COPY QSAV INTO Q AND GRAD INTO H, AND SET THE DIAGONAL ELEMENTS OF Q. PARQ 381
C THIS IS THE ENTRY POINT FOR SUBITERATIONS IN WHICH FLAMB IS PARQ 382
C INCREASED OR CONSTRAINTS ARE IMPOSED.    PARQ 383
C                                             PARQ 384
1850 KRANK=0                                 PARQ 385
      JQ=0                                   PARQ 386
      JT=0                                   PARQ 387
      DO 1940 JX=1,NV                       PARQ 388
1860 IF(MASK(JX))1940,1860,1940             PARQ 389
      JQ=JQ+1                               PARQ 390
      IF(MASKT(JX))1940,1870,1940          PARQ 391
1870 JT=JT+1                               PARQ 392
      H(JT)=GRAD(JQ)                       PARQ 393
      KQ=0                                   PARQ 394
      KT=0                                   PARQ 395
      DO 1930 KX=1,JX                       PARQ 396
1890 IF(MASK(KX))1930,1880,1930           PARQ 397
      KQ=KQ+1                               PARQ 398
      IF(MASKT(KX))1930,1890,1930         PARQ 399
1890 KT=KT+1                               PARQ 400
      SA=ERR(JQ,KQ)                         PARQ 401
      IF(KX-JX)1920,1900,1920             PARQ 402
1900 IF(SA)1910,1920,1910                 PARQ 403
1910 SA=RUNIT+FLAMB                        PARQ 404
      KRANK=KRANK+1                       PARQ 405
1920 ERR(KT,JT+1)=SA                      PARQ 406
1930 CONTINUE                              PARQ 407
1940 CONTINUE                              PARQ 408
C                                             PARQ 409
C SOLVE THE NORMAL EQUATIONS FOR H, THE CORRECTION VECTOR. PARQ 410
C THE METHOD USED IS GAUSSIAN ELIMINATION WITHOUT PIVOTING. PARQ 411
C (PIVOTING IS NOT NECESSARY FOR A POSITIVE DEFINITE MATRIX.) PARQ 412
C ONLY ABOUT N**3/6 MULTIPLICATIONS ARE DONE. PARQ 413
C THE CHOICE OF GAUSSIAN ELIMINATION RATHER THAN CHOLESKY PARQ 414
C DECOMPOSITION IS INTENTIONAL.           PARQ 415
C                                             PARQ 416
      NSMAL=0                               PARQ 417
      NMU=NACT-1                           PARQ 418
      IF(NMU)2100,2010,1950               PARQ 419
C                                             PARQ 420
C          REDUCE THE SYSTEM TO TRIANGULAR FORM, PARQ 420
C          UTILIZING THE SYMMETRY OF THE MATRIX. PARQ 421

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1950 DO 2000 J=1,NMU                        PARQ 422
      PIVOT=ERR(J,J+1)                    PARQ 423
      IF(PIVOT)1960,2000,1960            PARQ 424
1960 JPU=J+1                               PARQ 425
      DO 1990 K=JPU,NACT                  PARQ 426
      EM=ERR(J,K+1)/PIVOT                 PARQ 427
      IF(EM)1970,1990,1970               PARQ 428
1970 DO 1980 L=K,NACT                     PARQ 429
1980 ERR(L,K+1)=ERR(L,K+1)-ERR(J,L+1)*EM PARQ 430
1990 H(K)=H(K)-H(J)*EM                    PARQ 431
2000 CONTINUE                              PARQ 432
C                                             PARQ 433
C          DO THE BACK SOLUTION.           PARQ 434
2010 DO 2090 J=1,NACT                     PARQ 435
      J=(NACT+1)-JINV                    PARQ 436
      PIVOT=ERR(J,J+1)                    PARQ 437
      IF(PIVOT)2020,2020,2030            PARQ 438
2020 NSMAL=NSMAL+1                        PARQ 439
2030 IF(PIVOT)2050,2040,2050            PARQ 440
2040 H(J)=RZERO                           PARQ 441
      GO TO 2090                          PARQ 442
2050 SUM=RZERO                             PARQ 443
      IF(J=NACT)2060,2080,2080          PARQ 444
2060 JPU=J+1                               PARQ 445
      DO 2070 K=JPU,NACT                  PARQ 446
2070 SUM=SUM+ERR(J,K+1)*H(K)             PARQ 447
2080 H(J)=(H(J)-SUM)/PIVOT                PARQ 448
2090 CONTINUE                              PARQ 449
C                                             PARQ 450
2100 MRANK=NACT-NSMAL                     PARQ 451
C                                             PARQ 452
C IF THE COEFFICIENT MATRIX WAS RANK DEFICIENT, PRINT A MESSAGE. PARQ 453
C                                             PARQ 454
      IF(MRANK=NACT)2110,2180,2110        PARQ 455
2110 COSIN=HUGE                            PARQ 456
      IF(INTRPT)2140,2120,2120            PARQ 457
2120 WRITE(KW,2130)MRANK,NACT            PARQ 458
2130 FORMAT(/41H RANK-DEFICIENT NORMAL EQUATIONS IN MARQ.,9X,
      * 7H RANK =,I3,7X,18H ORDER OF MATRIX =,I3)
2140 IF(MRANK)2150,2150,2160             PARQ 460
2150 KFLAG=-4                              PARQ 461
      GO TO 3470                          PARQ 462
2160 IF(METHD)2180,2180,2170             PARQ 463
2170 IF(MRANK-KRANK)2770,2180,2180       PARQ 464
C                                             PARQ 465
C UNPACK AND DE-SCALE THE CORRECTION VECTOR H. PARQ 466
C COMPUTE THE INNER PRODUCTS SA, SB, AND SC. PARQ 467
C                                             PARQ 468
2180 SA=RZERO                              PARQ 469
      SB=RZERO                              PARQ 470
      SC=RZERO                              PARQ 471
      KX=NV                                PARQ 472
      KO=NACTV                             PARQ 473
      KT=NACT                              PARQ 474
      DO 2230 JX=1,NV                     PARQ 475
2190 IF(MASKT(KX))2190,2200,2190         PARQ 476
      HH=RZERO                              PARQ 477
2190 IF(MASK(KX))2220,2210,2220         PARQ 478
2200 HH=H(KT)                              PARQ 479
      SA=SA+HH*GRAD(KQ)                   PARQ 480
      SB=SB+HH*HH                          PARQ 481

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SC=SC+GRAD(KQ)*2
HH=HH+FMGN/SCALE(KQ)
KT=KT-1
2210 KQ=KQ-1
2220 H(KX)=HH
2230 KX=KX-1
C
C ADD THE CORRECTION VECTOR TO THE PARAMETER VECTOR AND
C INSURE THAT NO CONSTRAINTS ARE VIOLATED.
C THIS IS THE ENTRY POINT FOLLOWING A CUTSTEP.
C
2240 CONTINUE
IF(INTRMU)2270,2250,2250
2250 WRITE(KW,2260)(H(JX),JX=1,NV)
2260 FORMAT(/7X,14H CORRECTION = ,6E15.7/(21X,6E15.7))
2270 NACSV=NACT
FRMIN=RUNIT
NLOOP=0
JXLIM=0
2280 DO 2500 JX=1,NV
IF(MASKT(JX))2500,2290,2500
XSAV=XSAVE(JX)
XMX=XMAX(JX)
XMN=XMIN(JX)
HH=H(JX)
2300 IF(XSAV-XMX)2310,2300,2300
IF(HH)2310,2310,2330
2310 IF(XSAV-XMN)2320,2320,2360
2320 IF(HH)2330,2360,2360
2330 MASKT(JX)=1
NACT=NACT-1
X(JX)=XSAV
IF(INTRPU)2500,2340,2340
2340 WRITE(KW,2350)JX,XSAV
2350 FORMAT(/8H FIX X(I3,4H) = ,E12.5,
* 46H TEMPORARILY, TO AVOID VIOLATING A CONSTRAINT. )
GO TO 2500
2360 XLIM=XSAV+HH*FRMIN
IF(JX-JXLIM)2380,2370,2380
2370 IF(KBND)2410,2390,2390
2380 X(JX)=XLIM
IF(XLIM-XMX)2400,2400,2390
2390 X(JX)=XMX
JBND=1
GO TO 2420
2400 IF(XLIM-XMN)2410,2500,2500
2410 X(JX)=XMN
JBND=-1
2420 IF(JX-JXLIM)2460,2430,2460
2430 CONTINUE
IF(INTRPU)2500,2440,2440
2440 WRITE(KW,2450)JX,X(JX),FRMIN
2450 FORMAT(/26H CONSTRAINT VIOLATED BY X(I3,
* 20H). VALUE RESET TO ,E15.8,24H USING CUTSTEP FACTOR = ,
* E12.5 )
GO TO 2500
2460 IF(NLOOP)2500,2470,2500
2470 DENOM=XLIM-XSAV
IF(DENOM)2480,2500,2480
2480 FRAC=(X(JX)-XSAV)/DENOM

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IF(FRAC-FRMIN)2490,2500,2500
2490 FRMIN=FRAC
JXLIM=JX
KBND=JBND
2500 CONTINUE
C
C IF THE PROPOSED STEP WOULD VIOLATE ANY ALREADY ACTIVE CONSTRAINTS,
C FIX THOSE COMPONENTS OF H EQUAL TO ZERO AND RECOMPUTE THE
C OTHER COMPONENTS.
C
IF(NACT)2510,2510,2540
2510 KFLAG=3
IF(INTRPT)3470,2520,2520
2520 WRITE(KW,2530)
2530 FORMAT(///47H APPARENT CONSTRAINED OPTIMUM LIES IN A CORNER. )
GO TO 3470
2540 IF(NACT-NACSV)1850,2550,2550
2550 IF(NLOOP)2570,2560,2570
2560 NLOOP=1
IF(JXLIM)2280,2570,2280
2570 CONTINUE
IF(INTRMU)2600,2580,2580
2580 WRITE(KW,2590)(X(JX),JX=1,NV)
2590 FORMAT(/16X,5H X = ,6E15.7/(21X,6E15.7))
C
C CALCULATE THE NEW FITTED VALUES.
C
2600 CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHNEW)
NF=NF+1
IF(PHNEW-PHI)2910,2610,2650
C
C THE NEW VALUE OF PHI IS EXACTLY EQUAL TO THE OLD VALUE.
C CHECK FOR CONVERGENCE UNDER THE NFLAT OPTION.
C
2610 IF(NFLAT)2620,2910,2620
2620 KFLAG=2
IF(INTRPU)2730,2630,2630
2630 WRITE(KW,2640)
2640 FORMAT(///45H CONVERGENCE ACHIEVED UNDER THE NFLAT OPTION.)
GO TO 2730
C
C THE NEW VALUE OF PHI IS GREATER THAN THE OLD VALUE.
C
2650 CONTINUE
IF(INTRMU)2680,2660,2660
2660 WRITE(KW,2670)PHI,PHNEW
2670 FORMAT(/33X,11H OLD PHI = ,E15.8,4X,11H NEW PHI = ,E15.8)
C
C INSURE THAT JSUB HAS NOT EXCEEDED MXSUB.
C
2680 JSUB=JSUB+1
IF(JSUB-MXSUB)2690,2690,2700
2690 IF(METHD)2840,3240,2750
2700 KFLAG=-1
IF(INTRPU)2730,2710,2710
2710 WRITE(KW,2720)MXSUB
2720 FORMAT(///44H EXCEEDED MAXIMUM NUMBER OF SUBITERATIONS = ,IS,
* 9H IN MARQ.)
C
RESTORE X TO THE BASE POINT.
2730 DO 2740 JX=1,NV

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2740 X(JX)=XSAVE(JX)
      CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHI)
      GO TO 3470
C
C THE NEW FIT IS WORSE THAN THE OLD FIT. COMPUTE COSIN, THE COSINE
C OF THE ANGLE BETWEEN THE SCALED GRADIENT AND THE SCALED CORRECTION
C VECTOR.
C
2750 DENOM=SB*SC
      IF(DENOM)2770,2770,2760
2760 COSIN=SA/OSQRT(DENOM)
      IF(COSIN-CRIT)2770,2770,2840
C
C COSIN IS NOT GREATER THAN CRIT. INCREASE THE VALUE OF LAMBDA.
C
2770 UPFAC=FNU
      IF(METHD-1)2800,2780,2800
2790 IF(FLAMB*(FNU-RTMD)-RUNIT)2800,2800,2790
2790 UPFAC=RTMD+RUNIT/FLAMB
2800 FLAMB=FLAMB*UPFAC
      IF(NTRMU)2830,2810,2810
2810 WRITE(KM,2820)JSUB,COSIN,FLAMB
2820 FORMAT(/18H **** SUBITERATION,I3,4X,17H INCREASE LAMBDA.,
* 4X,14H COS(GAMMA) = ,E12.5,27X,9H LAMBDA =,E12.5)
2830 CONTINUE
C
C GO BACK AND FORM THE NORMAL EQUATIONS
C USING A LARGER VALUE OF LAMBDA.
C
      GO TO 1830
C
C COSIN IS GREATER THAN CRIT. CUT THE MAGNITUDE OF THE STEP, H.
C
2840 STFAC=STFAC/RTMD
      FLAMB=FLAMB*FLUPB
      IF(METHD)2850,2860,2860
2850 FMGN=FMGN/RTMD
2860 DO 2870 JX=1,NV
2870 H(JX)=(X(JX)-XSAVE(JX))/RTMD
      IF(NTRMU)2900,2880,2880
2880 WRITE(KM,2890)JSUB,COSIN,STFAC
2890 FORMAT(/18H **** SUBITERATION,I3,4X,16H TAKE CUT STEPS.,4X,
* 14H COS(GAMMA) = ,E12.5,4X,17H CUTSTEP FACTOR =,E12.5)
2900 CONTINUE
C
C GO TO 2240
C
C THE VALUE OF PHI HAS DECREASED. TRY A HALF STEP.
C
2910 IF(METHD)2930,3240,2920
2920 IF(METHD-2)2930,3240,2930
2930 DO 2980 JX=1,NV
      XTEMP(JX)=X(JX)
      IF(MASK(JX))2980,2940,2980
2940 X(JX)=XSAVE(JX)+(X(JX)-XSAVE(JX))/RTMD
      IF(X(JX)-XMAX(JX))2960,2960,2950
2950 X(JX)=XMAX(JX)
2960 IF(X(JX)-XMIN(JX))2970,2980,2980
2970 X(JX)=XMIN(JX)
2980 CONTINUE
      DO 2990 JPT=1,NPTS
2990 FITSV(JPT)=FIT(JPT)

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MARQ 562

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```

      CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHALF)
      NF=NF+1
C
C USE QUADRATIC INTERPOLATION, IN ORDER TO TRY TO REFINER THE
C POSITION OF THE MINIMUM OF PHI.
C
      RLFAC=RUNIT
      DENOM=RTMD*((PHNEW-PHALF)-(PHALF-PHI))
      IF(DENOM)3020,3020,3000
3000 STFAC=(PHI-PHNEW)/DENOM
      RSFAC=(RUNIT+STFAC)/RTMD
C
C DO NOT EXTRAPOLATE.
C
      IF(STFAC-RUNIT)3030,3020,3020
3020 STFAC=RZERO
3030 DO 3040 JX=1,NV
      XSAVE(JX)=X(JX)
3040 X(JX)=X(JX)+(XTEMP(JX)-X(JX))*STFAC
      IF(PHALF-PHNEW)3050,3100,3100
3050 RLFAC=RUNIT/RTMD
      JSUB=JSUB+1
      DO 3060 JX=1,NV
3060 XTEMP(JX)=XSAVE(JX)
      DO 3070 JPT=1,NPTS
3070 FITSV(JPT)=FIT(JPT)
      IF(NTRMU)3095,3080,3080
3080 WRITE(KM,3090)PHNEW,PHALF
3090 FORMAT(/21H HALF STEP SUCCEEDED.,15X,8H PHNEW =,E15.8,18X,
X 8H PHALF =,E15.8)
3095 PHNEW=PHALF
3100 IF(STFAC)3110,3120,3110
3110 CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHI)
      NF=NF+1
      IF(PHI-PHNEW)3180,3120,3120
3120 DO 3130 JX=1,NV
3130 X(JX)=XTEMP(JX)
      DO 3140 JPT=1,NPTS
3140 FIT(JPT)=FITSV(JPT)
      IF(STFAC)3150,3210,3150
3150 CONTINUE
      IF(NTRMU)3210,3160,3160
3160 WRITE(KM,3170)RSFAC,PHI
3170 FORMAT(/32H QUADRATIC INTERPOLATION FAILED.,15X,8H RSFAC =,E12.5,
X 12X,6H PHI =,E15.8)
      GO TO 3210
3180 RLFAC=RSFAC
      PHNEW=PHI
      IF(NTRMU)3210,3190,3190
3190 WRITE(KM,3200)RLFAC,PHI
3200 FORMAT(/35H QUADRATIC INTERPOLATION SUCCEEDED.,12X,8H RLFAC =,
* E12.5,12X,6H PHI =,E15.8)
3210 IF(RLFAC)3240,3240,3220
3220 FLAMB=FLAMB/RLFAC
      IF(METHD)3230,3240,3240
3230 FMGN=FMGN*RLFAC
C
C THE STEP IS ACCEPTED. TEST FOR CONVERGENCE IF NO CONSTRAINT
C BECAME ACTIVE DURING THIS ITERATION.
C
3240 CONTINUE

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MARQ 563
MARQ 564
MARQ 565
MARQ 566
MARQ 567
MARQ 568
MARQ 569
MARQ 570
MARQ 571
MARQ 572
MARQ 573
MARQ 574
MARQ 575
MARQ 577
MARQ 578
MARQ 579
MARQ 580
MARQ 581
MARQ 582
MARQ 584
MARQ 585
MARQ 586
MARQ 587
MARQ 588
MARQ 589
MARQ 590
MARQ 591
1000 591
2000 591
3000 591
MARQ 593
MARQ 594
MARQ 595
MARQ 596
MARQ 597
MARQ 598
MARQ 599
MARQ 700
MARQ 701
MARQ 702
MARQ 703
MARQ 704
1000 704
2000 704
MARQ 706
MARQ 707
MARQ 708
MARQ 709
MARQ 710
MARQ 711
MARQ 712
MARQ 713
MARQ 714
MARQ 715
MARQ 716
MARQ 717
MARQ 718
MARQ 719
MARQ 720
MARQ 721

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IF(NTRMU)3270,3250,3250 MARQ 722
3250 WRITE(KM,3260)ITER,PHNEW 1000 722
3260 FORMAT(/17H END ITERATION ,I4,58X,6H PHI =,E15.8) 2000 722
3270 PHI=PHNEW MARQ 726
IF(JXLM)3280,3280,3350 MARQ 727
3280 DO 3320 JX=1,NV MARQ 728
IF(MASK(JX))3320,3290,3320 MARQ 729
3290 DIF=XI(JX)-XSAVE(JX) MARQ 730
IF(DIF)3300,3310,3310 MARQ 731
3300 DIF=-DIF MARQ 732
3310 IF(DIF-DELMN(JX))3320,3320,3350 MARQ 733
3320 CONTINUE MARQ 734
KFLAG=1 MARQ 735
IF(NTRPU)3470,3330,3330 MARQ 736
3330 WRITE(KM,3340) MARQ 737
3340 FORMAT(/38H CONVERGED WHEN THE STEP BECAME SMALL.) MARQ 738
GO TO 3470 MARQ 739
C MARQ 740
C THE ITERATION HAS NOT YET CONVERGED. MARQ 741
C MARQ 742
3350 IF(ITER-MAXIT)3380,3380,3360 MARQ 743
3360 KFLAG=-6 MARQ 744
WRITE(KM,3370)MAXIT MARQ 745
3370 FORMAT(/47H MAXIMUM NUMBER OF ITERATIONS EXCEEDED IN MARQ.,5X, MARQ 746
* 9H MAXIT = ,I4 ) MARQ 747
GO TO 3470 MARQ 748
C MARQ 749
C IF SUBITERATIONS WERE NOT PERFORMED THIS ITERATION, DECREASE LAMBDA. MARQ 750
C MARQ 751
3380 IF(NF-NFMAX)3390,3390,3450 MARQ 752
3390 IF(JSUB)3400,3400,3440 MARQ 753
3400 FMGN=FMGN*RTMD MARQ 754
IF(FMGN-RUNIT)3420,3420,3410 MARQ 755
3410 FMGN=RUNIT MARQ 756
3420 SCALJ=RUNIT*FLAMB MARQ 757
IF(SCALJ-RUNIT)3440,3440,3430 MARQ 758
3430 FLAMB=FLAMB/FMU MARQ 759
3440 CONTINUE MARQ 760
C MARQ 761
GO BACK AND DO ANOTHER ITERATION. MARQ 762
GO TO 1420 MARQ 763
3450 KFLAG=-7 MARQ 764
WRITE(KM,3460)NFMAX MARQ 765
3460 FORMAT(/25H NF HAS EXCEEDED NFMAX = ,I7,9H IN MARQ. ) MARQ 766
C MARQ 767
C ***** MARQ 768
C MARQ 769
C THE ITERATION HAS TERMINATED. MARQ 770
C PRINT OUT THE DATA, FITTED VALUES, AND RESIDUALS. MARQ 771
C COMPUTE AND PRINT THE STANDARD DEVIATION OF THE DATA FROM THE FIT. MARQ 772
C MARQ 773
3470 CONTINUE MARQ 774
IF(NTRPU)3670,3480,3480 MARQ 775
3480 WRITE(KM,3490)ITER,NF,PHI,FMGN,FLAMB MARQ 776
3490 FORMAT(/1X,I4,11H ITERATIONS,7X,5H NF =,I5,9X,6H PHI =,E15.8, MARQ 777
X 10X,7H FMGN =,E12.5,7X,9H LAMBDA =,E12.5) 1000 775
WRITE(KM,2590)(X(JX),JX=1,NV) MARQ 778
IF(NTRAC)3520,3500,3500 MARQ 779
3500 WRITE(KM,3510) MARQ 780
3510 FORMAT(/14X,14H J,9X,5H Y(J),14X,7H FIT(J),10X, MARQ 781
* 12H Y(J)-FIT(J),7X,8H YSIG(J),11X,13H (Y-FIT)/YSIG/1H ) MARQ 782

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3520 SIG=YSIG(1) MARQ 783
RMSDV=RZERO MARQ 784
SDVMX=RZERO MARQ 785
DO 3610 JPT=1,NPTS MARQ 786
IF(LEQU)3540,3530,3540 MARQ 787
3530 SIG=YSIG(JPT) MARQ 788
3540 YY=Y(JPT) MARQ 789
PTERM=YY-FIT(JPT) MARQ 790
RTERM=PTERM/SIG MARQ 791
IF(NTRAC)3570,3550,3550 MARQ 792
3550 WRITE(KM,3560)JPT,YY,FIT(JPT),PTERM,SIG,RTERM MARQ 793
3560 FCRMAT(10X,I5,5X,E15.8,5X,E15.8,5X,E12.5,5X,E12.5,10X,E12.5) MARQ 794
3570 RMSDV=RMSDV+RTERM**2 MARQ 795
IF(RTERM)3580,3590,3590 MARQ 796
3580 RTERM=-RTERM MARQ 797
3590 IF(RTERM-SDVMX)3610,3610,3600 MARQ 798
3600 SDVMX=RTERM MARQ 799
3610 CONTINUE MARQ 900
DENOM=NPTS*NACTV MARQ 801
WRITE(KM,3620)DENOM MARQ 802
3620 FORMAT(/32H NUMBER OF DEGREES OF FREEDOM = ,E12.5 ) MARQ 803
IF(DENOM)3650,3650,3630 MARQ 804
3630 RMSDV=QSQRTRMSDV/DENOM MARQ 805
WRITE(KM,3640)RMSDV MARQ 806
3640 FORMAT(/43H R.M.S. SCALED DEVIATION OF DATA FROM FIT =,E12.5) MARQ 807
3650 CONTINUE MARQ 808
WRITE(KM,3660)SDVMX MARQ 809
3660 FORMAT(/27H MAXIMUM SCALED DEVIATION =,E12.5) MARQ 810
C MARQ 811
C CALL FUNC TO SET THE FINAL VALUES. MARQ 812
C MARQ 813
3670 CALL FUNC (FUNK,Y,YSIG,NPTS,FIT,PHI) MARQ 814
FOBJ=PHI MARQ 815
C MARQ 816
C CALL MQERR TO PRINT THE PARAMETER ERRORS AND CORRELATIONS. MARQ 817
C A DUMMY ROUTINE MAY BE SUBSTITUTED FOR MQERR IF THESE ARE NOT NEEDED. MARQ 818
C MARQ 819
IF(MATRX)3690,3690,3680 MARQ 820
3690 CALL MQERR (NACTV,H,SCALE,NPTS) MARQ 821
C MARQ 822
3690 RETURN MARQ 823
C END MARQ. MARQ 824
END MARQ 825
C MARQ 826
SUBROUTINE FUNC (FUNK,Y,YSIG,NPTS,FIT,PHI) MARQ 827
C MARQ 828
C FUNC CALLS FUNK OR FOFX TO COMPUTE THE ARRAY OF FITTED VALUES FITM. MARQ 829
C MARQ 830
C DOUBLE PRECISION X,FIT,F,PHI,SIG,YY MARQ 831
C MARQ 832
C DIMENSION Y(1),YSIG(1),FIT(1) MARQ 833
C MARQ 834
COMMON /CSTEP/ X(20),XMAX(20),XMIN(20),DELTX(20),DELMN(20), MARQ 835
* ERR(20,21),FOBJ,NV,NTRAC,MATRX,MASK(20), MARQ 836
* NFMAX,NFLAT,JVARY,NXTRA,KFLAG,NOREP,KERFL,KM MARQ 837
COMMON /NLS2/FLAMB,RELOF,METHD,KALCP,KORDF,MAXIT,LEQU,NXSUB,MXUPDFUNC MARQ 838
C MARQ 839
RZERO=0. MARQ 840
IF(KALCP)5050,5040,5050 MARQ 841
C MARQ 842

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5040 CALL FUNK (FIT)
      GO TO 5070
C
5050 DO 5060 JPT=1,NPTS
      CALL FOFX(JPT,NV,X,F)
5060 FIT(JPT)=F
C
5070 PHI=RZERO
      SIG=YSIG(1)
      DO 5130 JPT=1,NPTS
          IF(LEQU)5090,5080,5090
5080 SIG=YSIG(JPT)
C
          CHECK FOR AN ILLEGAL VALUE OF SIG.
5090 IF(SIG)5100,5100,5120
5100 WRITE(KM,5110)LEQU,JPT,SIG
5110 FORMAT(/28H ERROR IN MARQ.... LEQU = ,I1,5X,6HJPT = ,I5,5X,
* 7HYSIG = ,E12,5,14H IS .LE. ZERO.)
      STOP
C
5120 YY=Y(JPT)
5130 PHI=PHI+((FIT(JPT)-YY)/SIG)**2
C
      RETURN
C END FUNC.
      END
C
      SUBROUTINE DERIV (JPT,FUNK,NPTS,FIT,FITSV,P,LPCOL)
C
C DERIV A.0 A.N.S.I. STANDARD FORTRAN JUNE 1975
C
C DERIV COMPUTES THE JACOBIAN MATRIX P USING FINITE DIFFERENCES.
C
C P(J,K) IS THE PARTIAL DERIVATIVE OF FIT(J) WITH RESPECT TO X(K).
C IF KORDF.EQ.1, DERIV USES A NONCENTRAL DIFFERENCE FORMULA.
C IF KORDF.EQ.2, DERIV USES A CENTRAL DIFFERENCE FORMULA.
C KORDF.EQ.1 IS ABOUT TWICE AS FAST AS KORDF.EQ.2, BUT LESS ACCURATE.
C ON A MACHINE HAVING LESS THAN ABOUT TEN SIGNIFICANT DIGITS IN
C SINGLE PRECISION (FOR EXAMPLE THE IBM 360 OR 370), THE
C DIFFERENCING SHOULD BE DONE IN DOUBLE PRECISION. TO ACCOMPLISH
C THIS, ACTIVATE THE DOUBLE PRECISION STATEMENT BELOW.
C
      DOUBLE PRECISION X,FIT,FITSV,DEL,TMODL,XSAVE,FX0,FX1
C
      DIMENSION FIT(1),FITSV(1),P(LPCOL,1)
C
      COMMON /CSTEP/ X(20),XMAX(20),XMIN(20),DELTX(20),DELMNI(20),
* ERR(20,21),FOBJ,NV,NTRAC,MATRIX,MASK(20),
* NFMX,NFLAT,JVARY,NXTRA,KFLAG,NOREP,KERFL,KM
      COMMON //LLS2/FLAMB,RELDF,METHD,KALCP,KORDF,MAXIT,LEQU,MXSUB,MXUPD1000V
C
      JVARY=0
C
C SAVE FIT IF KALCP.GE.0 .
      IF(KALCP)6020,6000,6000
6000 DO 6010 J=1,NPTS
6010 FITSV(J)=FIT(J)
C
C LOOP OVER THE ACTIVE PARAMETERS X(JX).
C

```

```

FUNC 23
FUNC 24
FUNC 25
FUNC 26
FUNC 27
FUNC 28
FUNC 29
FUNC 30
FUNC 31
FUNC 32
FUNC 33
FUNC 34
FUNC 35
FUNC 36
FUNC 37
FUNC 38
FUNC 39
FUNC 40
FUNC 41
FUNC 42
FUNC 43
FUNC 44
FUNC 45
FUNC 46
FUNC 47
FUNC 48
DERIV 1
DERIV 2
DERIV 3
DERIV 4
DERIV 5
DERIV 6
DERIV 7
DERIV 8
DERIV 9
DERIV 10
DERIV 11
DERIV 12
DERIV 13
DERIV 14
DERIV 15
DERIV 16
DERIV 17
DERIV 18
DERIV 19
DERIV 20
DERIV 21
DERIV 22
DERIV 23
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DERIV 27
DERIV 28
DERIV 29
DERIV 30
DERIV 31
DERIV 32
DERIV 33
DERIV 34
DERIV 35

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6020 KX=0
      DO 6210 JX=1,NV
          IF(MASK(JX))6210,6030,6210
6030 KX=KX+1
          DEL=RELD*X(JX)
          IF(DEL)6050,6040,6050
6040 DEL=RELD
6050 XSAVE=X(JX)
          X(JX)=XSAVE+DEL
          TMODL=DEL+DEL
          IF(KALCP)6160,6060,6120
C
C KALCP.EQ.0 . COMPUTE P, ONE COLUMN AT A TIME.
C
6060 CALL FUNK (FIT)
          IF(KORDF-2)6070,6090,6240
6070 DO 6080 J=1,NPTS
6080 P(J,KX)=(FIT(J)-FITSV(J))/DEL
          GO TO 6200
C
C KALCP.EQ.0 AND KORDF.EQ.2 . IN THIS CASE, THE INPUT VALUES OF
C FIT(J) WILL BE DESTROYED.
C
6090 X(JX)=XSAVE-DEL
          DO 6100 J=1,NPTS
              FITSV(J)=FIT(J)
              JVARY=XJ
              CALL FUNK (FIT)
              JVARY=0
              DO 6110 J=1,NPTS
                  P(J,KX)=(FITSV(J)-FIT(J))/TMODL
                  GO TO 6200
C
C KALCP.GT.0 . COMPUTE P, ONE ELEMENT AT A TIME.
C
6120 DO 6150 J=1,NPTS
          CALL FOFX (J,NV,X,FX1)
          IF(KORDF-2)6130,6140,6240
6130 P(J,KX)=(FX1-FITSV(J))/DEL
          GO TO 6150
6140 X(JX)=XSAVE-DEL
          CALL FOFX (J,NV,X,FX0)
          P(J,KX)=(FX1-FX0)/TMODL
          X(JX)=XSAVE+DEL
6150 CONTINUE
          GO TO 6200
C
C KALCP.LT.0 . COMPUTE ONE ROW OF P, ONE ELEMENT AT A TIME.
C
6160 FITSV(1)=FIT(JPT)
          CALL FOFX (JPT,NV,X,FX1)
          IF(KORDF-2)6170,6180,6170
6170 P(1,KX)=(FX1-FITSV(1))/DEL
          GO TO 6190
6180 X(JX)=XSAVE-DEL
          CALL FOFX (JPT,NV,X,FX0)
          P(1,KX)=(FX1-FX0)/TMODL
6190 FIT(JPT)=FITSV(1)
6200 X(JX)=XSAVE
6210 CONTINUE
          DERIV 36
          DERIV 37
          DERIV 38
          DERIV 39
          DERIV 40
          DERIV 41
          DERIV 42
          DERIV 43
          DERIV 44
          DERIV 45
          DERIV 46
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          DERIV 48
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          DERIV 88
          DERIV 89
          DERIV 90
          DERIV 91
          DERIV 92
          DERIV 93
          DERIV 94
          DERIV 95

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C      IF(KALCP)6240,6220,6220
6220 DD 6230 J=1,NPTS
6230 FIT(IJ)=FITSV(J)
C
6240 RETURN
C END DERIV.
      END
C
      SUBROUTINE MQERR (NACTV,H,SCALE,NPTS)
C
C      MQERR 1.3      A.N.S.I. STANDARD FORTRAN      FEBRUARY 1978
C
C      MQERR IS CALLED BY MARO TO COMPUTE AND PRINT APPROXIMATE VALUES OF
C      THE PARAMETER ERRORS AND CORRELATIONS.
C
C      FOR THE MEANING OF THE -MAXIMUM VARIANCE INFLATION FACTOR- BELOW,
C      SEE...      D. W. MARQUARDT AND R. D. SNEE,
C      RIDGE REGRESSION IN PRACTICE,
C      THE AMERICAN STATISTICIAN 29 (1975) 3-20
C
C      INPUT QUANTITIES.....  KM,ERP,NACTV,SCALE,NPTS,NV,NTRAC,MASK,FOBJ
C      OUTPUT QUANTITIES....  ERR
C      SCRATCH STORAGE.....   H
C
      DOUBLE PRECISION X
C
      DIMENSION H(1),SCALE(1)
C
      COMMON /CSTEP/ X(20),XMAX(20),XMIN(20),DELTX(20),DELMN(20),
*      ERRI(20,21),FOBJ,NV,NTRAC,MATRX,MASK(20),
*      NF*AX,NFLAT,JVARY,NXTRA,KFLAG,NOREP,KERFL,Kb
C
      QSQRT(ARG)=SQRT(ARG)
C
*****
C
      RZERO=0.
      RUNIT=1.
      HUGE=1.E30
C
      PRINT QSAV.
C
      IF(INTRAC=(-1))4040,4000,4000
4000 WRITE(KW,4010)
4010 FORMAT(///18H SUBROUTINE MQERR.//26H QSAV (PT*P, SCALED, WHERE,
*      23H P IS THE JACOBIAN).....)
      DD 4030 JX=1,NACTV
      WRITE(KW,4020)JX,(ERR(JX,KX),KX=1,JX)
4020 FORMAT(//1X,I3,2X,6E15.7//16X,6E15.7))
4030 CONTINUE
C
C      COMPUTE THE SCALED ERROR MATRIX, WHICH IS THE INVERSE OF QSAV.
C      INVERT QSAV USING THE GAUSS-JORDAN METHOD WITHOUT PIVOTING.
C      (PIVOTING IS NOT NECESSARY FOR A POSITIVE DEFINITE MATRIX.)
C      ONLY ABOUT N**3/2 MULTIPLICATIONS ARE DONE.
C      F. L. BAUER AND C. REINSCH, P. 45 IN -LINEAR ALGEBRA-
C      BY J. H. WILKINSON AND C. REINSCH (SPRINGER-VERLAG, 1971)
C      THE INVERSE IS NOT GUARANTEED TO BE POSITIVE DEFINITE, DUE TO
C      ROUND-OFF ERROR. NEVERTHELESS, THE CHOICE OF THE GAUSS-JORDAN METHOD
C      RATHER THAN CHOLESKY DECOMPOSITION IS INTENTIONAL.
C      HI(*) IS USED AS A SCRATCH VECTOR.

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DFRIV 96
DFRIV 97
DERIV 98
DERIV 99
DERIV100
DERIV101
DERIV102
DERIV103
DERIV104
MQERR 1
MQERR 2
MQERR 3
MQERR 4
MQERR 5
MQERR 6
MQERR 7
MQERR 8
MQERR 9
MQERR 10
MQERR 11
MQERR 12
MQERR 13
MQERR 14
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MQERR 40
MQERR 41
MQERR 42
MQERR 43
MQERR 44
MQERR 45
MQERR 46
MQERR 47
MQERR 48
MQERR 49
MQERR 50
MQERR 51

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C
4040 NSMAL=0
      DD 4200 LINV=1,NACTV
      L=(NACTV+1)-LINV
      PIVOT=ERR(1,1)
      IF(PIVOT)4050,4050,4060
4050 NSMAL=NSMAL+1
4060 IF(NACTV-2)4140,4070,4070
4070 DD 4130 K=2,NACTV
      Q=ERR(K,1)
      IF(PIVOT)4090,4080,4090
      H(K)=RZERO
      GO TO 4120
4090 IF(K-L)4100,4100,4110
4100 H(K)=-Q/PIVOT
      GO TO 4120
4110 H(K)=Q/PIVOT
4120 DD 4130 M=2,K
4130 ERR(K-1,M-1)=ERP(K,M)+Q*H(M)
4140 IF(PIVOT)4150,4150,4150
4150 ERR(NACTV,NACTV)=RZERO
      GO TO 4170
4150 ERR(NACTV,NACTV)=RUNIT/PIVOT
4170 IF(NACTV-2)4200,4180,4180
4180 DC 4190 K=2,NACTV
4190 ERP(NACTV,K-1)=H(K)
4200 CONTINUE
C
      NPANK=NACTV-NSPAL
      IF(NRANK=NACTV)4210,4230,4230
4210 CONTINUE
      WRITE(KW,4220)NRANK,NACTV
4220 FORMAT(//44H THE SECOND DERIVATIVE MATRIX IS SINGULAR IN,
*      7H MQERR.,9X,7H RANK =,I3,7X,8H ORDER =,I3//
*      45H THEREFORE ALL PARAMETER ERRORS ARE INFINITE. )
C
      UNPACK THE ERROR MATRIX INTO THE UPPER
      TRIANGLE OF ERR(*,*), DE-SCALING IT.
C
4230 JV=0
      VIFNX=RZERO
      DD 4380 JX=1,NV
      IF(MASK(JX))4250,4240,4250
4240 JV=JV+1
4250 KV=0
      DD 4370 KX=1,JX
      ER=RZERO
      IF(MASK(JX))4370,4260,4370
4260 IF(MASK(KX))4370,4270,4370
4270 KV=KV+1
      TEMP=RZERO
      DENOM=SCALE(JV)*SCALE(KV)
      IF(DENOM)4280,4290,4280
4280 TEMP=ERR(JV,KV)
      ER=TEMP/DENOM
4290 IF(JX-KX)4370,4300,4370
4300 IF(TEMP)4310,4310,4350
4310 CONTINUE
      IF(INTRAC=(-2))4340,4320,4320
4320 WRITE(KW,4330)JX,KX,TEMP
4330 FORMAT(//6H THE (I3,I3,1H,,I3,23H) ELEMENT OF QSAV***-1 =,E12.5,

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MQERR 52
MQERR 53
MQERR 54
MQERR 55
MQERR 56
MQERR 57
MQERR 58
MQERR 59
MQERR 60
MQERR 61
MQERR 62
MQERR 63
MQERR 64
MQERR 65
MQERR 66
MQERR 67
MQERR 68
MQERR 69
MQERR 70
MQERR 71
MQERR 72
MQERR 73
MQERR 74
MQERR 75
MQERR 76
MQERR 77
MQERR 78
MQERR 79
MQERR 80
MQERR 81
MQERR 82
MQERR 83
MQERR 84
1000R 84
MQERR 85
MQERR 86
MQERR 87
MQERR 88
MQERR 89
MQERR 90
MQERR 91
MQERR 92
MQERR 93
MQERR 94
MQERR 95
MQERR 96
MQERR 97
MQERR 98
MQERR 99
MQERR100
MQERR101
MQERR102
MQERR103
MQERR104
MQERR105
MQERR106
MQERR107
MQERR108
MQERR109
MQERR110
MQERR111

```

```

*      5X,45H THEREFORE ALL PARAMETER ERRORS ARE INFINITE. )
4340  TEMP=-TEMP
4350  IF(TEMP-VIFMX)4370,4370,4360
4360  VIFX=TEMP
4370  ERR(KX,JX+1)=ER
4380  CONTINUE
C      SYMMETRIZE ERR.
      DD 4390 JX=1,NV
      DD 4390 KX=1,JX
      ERR(JX,KX)=ERR(KX,JX+1)
4390  ERR(KX,JX)=ERR(JX,KX)
C
C      COMPUTE AND PRINT THE STANDARD ERRORS.
      NDF=NPTS-NACTV
      SCFAC=HUGE
      IF(NDF)4410,4410,4400
4400  SCFAC=NCF
      SCFAC=QSQRT(FOBJ/SCFAC)
4410  RESCL=NDF*NDF
      IF(RESCL)4430,4430,4420
4420  RESCL=QSQRT(RESCL)
4430  CONTINUE
      IF(ATRAC-(1))4520,4440,4440
4440  WRITE(KW,4450)NDF,NDF,RESCL,FOBJ,SCFAC
4450  FORMAT(///41H NUMBER OF DEGREES OF FREEDOM (N.D.F.) = ,
*      15H(NPTS-NACTV) = ,15//24H EXPECTED VALUE OF PHI = ,
*      39H N.D.F. PLUS OR MINUS SQRT(2*N.D.F.) = ,15,
*      16H PLUS OR MINUS ,E12.5//23H ACTUAL VALUE OF PHI = ,E12.5
*      //39H RESCALING FACTOR = SQRT(PHI/N.D.F.) = ,E12.5 )
      WRITE(KW,4460)VIFMX
4460  FORMAT(//36H MAXIMUM VARIANCE INFLATION FACTOR =,E12.5//
*      32H APPROXIMATE STANDARD ERRORS....//68X,9H RESCALED/
*      12X,1HJ,6X,7HMASK(J),9X,4HX(J),14X,5HERROR,12X,5HERROR )
      RESCL=HUGE
      DD 4540 JX=1,NV
      SCALJ=RUNIT
      ER=ERR(JX,JX)
      IF(ER)4470,4490,4480
4470  ER=QSQRT(-ER)
      SCALJ=ER
      GO TO 4490
4480  ER=QSQRT(ER)
      SCALJ=ER
4490  IF(NDF)4520,4520,4500
4500  IF(NRANK-NACTV)4520,4510,4510
4510  RESCL=SCFAC*ER
4520  SCALE(JX)=SCALJ
      WRITE(KW,4530)JX,MASK(JX),X(JX),ER,RESCL
4530  FORMAT(/10X,I3,I10,6X,E16.8,4X,E13.5,4X,E13.5 )
4540  CONTINUE
C      COMPUTE AND PRINT THE CORRELATIONS.
      IF(NV-2)4620,4550,4550
4550  CONTINUE
      WRITE(KW,4560)(K,K=1,NV)
4560  FORMAT(///45H LOWER TRIANGLE OF THE CORRELATION MATRIX....//
*      12X,7H K .....I5,7I13/(11X,8I13))
      WRITE(KW,4570)(MASK(K),K=1,NV)
4570  FORMAT(/7X,12H MASK(K).....I5,7I13/(11X,8I13))
      WRITE(KW,4580)
4580  FORMAT(/3X,1HJ,4X,7HMASK(J) )

```

```

YQERR112
YQERR113
YQERR114
YQERR115
YQERR116
YQERR117
YQERR118
YQERR119
YQERR120
YQERR121
YQERR122
YQERR123
YQERR124
YQERR125
YQERR126
YQERR127
YQERR128
YQERR129
YQERR130
YQERR131
YQERR132
YQERR133
YQERR134
YQERR135
YQERR136
YQERR137
YQERR138
YQERR140
YQERR141
YQERR142
YQERR143
YQERR144
YQERR145
YQERR146
YQERR147
YQERR148
YQERR149
YQERR150
YQERR151
YQERR152
YQERR153
YQERR154
YQERR155
YQERR156
YQERR157
YQERR158
YQERR159
YQERR160
YQERR161
YQERR162
YQERR163
YQERR164
YQERR165
YQERR166
YQERR167
YQERR168
YQERR169
YQERR170
YQERR171
DD 4610 JX=1,NV
DD 4590 KX=1,JX
4590  H(KX)=ERR(JX,KX)/(SCALE(JX)*SCALE(KX))
      WRITE(KW,4600)JX,MASK(JX),(H(KX),KX=1,JX)
4600  FORMAT(/1X,I3,I8,5X,8E13.5/(17X,8E13.5))
4610  CONTINUE
C
C      END MDERR.
4620  RETURN
      END
C

```

```

YQERR172
YQERR173
YQERR174
YQERR175
YQERR176
YQERR177
YQERR178
YQERR180
YQERR179
YQERR181
YQERR182

```



```

C      PR....PROBLEM NUMBER (MAY BE ALPHANUMERIC)
C      N.....NUMBER OF OBSERVATIONS
C      M.....HIGHEST DEGREE POLYNOMIAL SPECIFIED
C      NDT...TEMPERATURE DIFFERENCE
C      TM....STARTING TEMPERATURE
C
C      POINT PROBLEM PARAMETERS.
C
C      WRITE (6,3) PR
C      WRITE (6,4) N
C      WRITE (6,16) M
C      WRITE (6,18) NDT
C      WRITE (6,17) DTEMP
C
C      READ INPUT DATA
C
C      L=N*M
C      DO 110 I=1,N
C      J=L+I
C
C      X(I) IS THE INDEPENDENT VARIABLE, AND X(J) IS THE DEPENDENT
C      VARIABLE.
C
C      X(I)=TEMP(I)
C      IF(IKK .EQ. 1) X(J)=AA1(I)
C      IF(IKK .EQ. 2) X(J)=AA2(I)
C      IF(IKK .EQ. 3) X(J)=AA3(I)
C      IF(IKK .EQ. 4) X(J)=BB1(I)
C      IF(IKK .EQ. 5) X(J)=BB2(I)
110 CONTINUE
C
C      CALL GDATA (N,M,X,XBAR,STD,D,SUMSQ)
C
C      MM=M+1
C      SUM=0.0
C      NT=N-1
C
C      DO 200 I=1,M
C      ISAVE(I)=I
C
C      FORM SUBSET OF CORRELATION COEFFICIENT MATRIX
C
C      CALL ORDER (MM,D,MM,I,ISAVE,DI,E)
C
C      INVERT THE SUBMATRIX OF CORRELATION COEFFICIENTS
C
C      CALL MINV (DI,I,DET,B,T)
C
C      CALL MULTR (N,I,XBAR,STD,SUMSQ,DI,E,ISAVE,B,SB,T,ANS)
C
C      PRINT THE RESULT OF CALCULATION
C
C      WRITE (6,5) I
C      SUMIP=ANS(4)-SUM
C      IF(SUMIP) 140, 140, 150
140 WRITE (6,13)
C      GO TO 210
150 WRITE (6,6) ANS(1)
C      WRITE (6,7) (B(J),J=1,I)
C      WRITE (6,8) I

```

```

POLRG120
POLRG121
POLRG122
POLRG123
POLRG124
POLRG125
POLRG126
POLRG127
POLRG128
POLRG129
POLRG130
POLRG131
POLRG132
POLRG133
POLRG134
POLRG135
POLRG136
POLRG137
POLRG138
POLRG139
POLRG140
POLRG141
POLRG142
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POLRG160
POLRG161
POLRG162
POLRG163
POLRG164
POLRG165
POLRG166
POLRG167
POLRG168
POLRG169
POLRG170
POLRG171
POLRG172
POLRG173
POLRG174
POLRG175
POLRG176
POLRG177
POLRG178
POLRG179

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```

      WRITE (6,9)
      SUM=ANS(4)
      WRITE (6,10) I,ANS(4),ANS(6),ANS(10),SUMIP
      NI=ANS(8)
      WRITE (6,11) NI,ANS(7),ANS(9)
      WRITE (6,12) NT,SUMSQ(MM)
C
C      SAVE COEFFICIENTS FOR CALCULATION OF Y ESTIMATES
C
C      CCE(I)=ANS(1)
C      DO 160 J=1,I
160 CDE(J+1)=B(IJ)
C      LA=I
C      200 CONTINUE
C
C      CALCULATE ESTIMATES
C
C      NP3=N+N
C      DO 230 I=1,N
C      NP3=NP3+1
C      P(NP3)=CCE(I)
C      L=I
C      DO 230 J=1,LA
C      P(NP3)=P(NP3)+X(I)*CDE(J+1)
230 L=L+M
C
C      COPY OBSERVED DATA
C
C      N2=N
C      L=N*M
C      DO 240 I=1,N
C      P(I)=X(I)
C      N2=N2+1
C      L=L+1
240 P(N2)=X(L)
C
C      PRINT TABLE OF RESIDUALS
C
C      WRITE (6,3) PR
C      WRITE (6,5) LA
C      WRITE (6,14)
C      NP2=N
C      NP3=N+N
C      DO 250 I=1,N
C      NP2=NP2+1
C      NP3=NP3+1
C      RESID=P(NP2)-P(NP3)
C      PRERR=RESID*100./P(NP2)
250 WRITE (6,15) I,P(I),P(NP2),P(NP3),RESID,PRERR
C
C      300 CONTINUE
C      STOP
C      END
C
C      .....
C      SUBROUTINE MINV
C
C      PURPOSE

```

```

POLRG180
POLRG181
POLRG182
POLRG183
POLRG184
POLRG185
POLRG186
POLRG187
POLRG188
POLRG189
POLRG190
POLRG191
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POLRG193
POLRG194
POLRG195
POLRG196
POLRG197
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POLRG220
POLRG221
POLRG222
POLRG223
POLRG224
POLRG225
POLRG226
POLRG227
POLRG228
POLRG229
POLRG230
POLRG231
POLRG232
POLRG233
FINV 001
FINV 002
MINV 003
MINV 004
FINV 005
MINV 006

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```

C          INVERT A MATRIX                                MINV 007
C          USAGE                                         MINV 008
C          CALL MINV(A,N,D,L,M)                          MINV 009
C
C          DESCRIPTION OF PARAMETERS                    MINV 010
C          A - INPUT MATRIX, DESTROYED IN COMPUTATION AND REPLACED BY MINV 011
C          RESULTANT INVERSE.                            MINV 012
C          N - ORDER OF MATRIX A                        MINV 013
C          D - RESULTANT DETERMINANT                     MINV 014
C          L - WORK VECTOR OF LENGTH N                   MINV 015
C          M - WORK VECTOR OF LENGTH N                   MINV 016
C
C          REMARKS                                       MINV 017
C          MATRIX A MUST BE A GENERAL MATRIX             MINV 018
C
C          SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED MINV 019
C          NDNE                                          MINV 020
C
C          METHOD                                         MINV 021
C          THE STANDARD GAUSS-JORDAN METHOD IS USED. THE DETERMINANT MINV 022
C          IS ALSO CALCULATED. A DETERMINANT OF ZERO INDICATES THAT MINV 023
C          THE MATRIX IS SINGULAR.                       MINV 024
C
C          .....                                       MINV 025
C
C          SUBROUTINE MINV(A,N,D,L,M)                    MINV 026
C          DIMENSION A(1),L(1),M(1)                     MINV 027
C
C          .....                                       MINV 028
C          IF A DOUBLE PRECISION VERSION OF THIS ROUTINE IS DESIRED, THE MINV 029
C          C IN COLUMN 1 SHOULD BE REMOVED FROM THE DOUBLE PRECISION MINV 030
C          STATEMENT WHICH FOLLOWS.                     MINV 031
C
C          DOUBLE PRECISION A,D,BIGA,HOLD,DABS          MINV 032
C
C          THE C MUST ALSO BE REMOVED FROM DOUBLE PRECISION STATEMENTS MINV 033
C          APPEARING IN OTHER ROUTINES USED IN CONJUNCTION WITH THIS MINV 034
C          ROUTINE.                                      MINV 035
C
C          THE DOUBLE PRECISION VERSION OF THIS SUBROUTINE MUST ALSO MINV 036
C          CONTAIN DOUBLE PRECISION FORTRAN FUNCTIONS. ABS IN STATEMENT MINV 037
C          10 MUST BE CHANGED TO DABS.                   MINV 038
C
C          .....                                       MINV 039
C          SEARCH FOR LARGEST ELEMENT                     MINV 040
C
C          D=1.0                                         MINV 041
C          NK=-N                                         MINV 042
C          DO 50 K=1,N                                   MINV 043
C          NK=NK+N                                       MINV 044
C          L(K)=K                                         MINV 045
C          M(K)=K                                         MINV 046
C          KK=NK*K                                       MINV 047
C          BIGA=A(KK)                                     MINV 048
C          DO 20 J=K,N                                   MINV 049
C          IZ=N*(J-1)                                    MINV 050
C          DO 20 I=K,N                                   MINV 051
C
C          IJ=I+J                                       MINV 052
C          10 IF(DABS(BIGA)-DABS(A(IJ))) 15,20,20       MINV 053
C          BIGA=A(IJ)                                     MINV 054
C          L(K)=I                                       MINV 055
C          M(K)=J                                       MINV 056
C          20 CONTINUE                                   MINV 057
C
C          INTERCHANGE ROWS                               MINV 058
C
C          J=L(K)                                        MINV 059
C          IF(J-K) 35,35,25                               MINV 060
C          KI=K-N                                         MINV 061
C          DO 30 I=1,N                                    MINV 062
C          KI=KI+N                                       MINV 063
C          HOLD=-A(KI)                                    MINV 064
C          JI=KI-K+J                                     MINV 065
C          A(KI)=A(JI)                                   MINV 066
C          30 A(JI)=HOLD                                  MINV 067
C
C          INTERCHANGE COLUMNS                           MINV 068
C
C          75 I=M(K)                                     MINV 069
C          IF(I-K) 45,45,38                               MINV 070
C          JP=N*(I-1)                                    MINV 071
C          DO 40 J=1,N                                   MINV 072
C          JK=NK+J                                       MINV 073
C          JI=JP+J                                       MINV 074
C          HOLD=-A(JK)                                   MINV 075
C          A(JK)=A(JI)                                   MINV 076
C          40 A(JI)=HOLD                                  MINV 077
C
C          DIVIDE COLUMN BY MINUS PIVOT, (VALUE OF PIVOT ELEMENT IS MINV 078
C          CONTAINED IN BIGA)                             MINV 079
C
C          45 IF(BIGA) 48,46,48                           MINV 080
C          46 D=0.0                                       MINV 081
C          RETURN                                         MINV 082
C          48 DO 55 I=1,N                                 MINV 083
C          IF(I-K) 50,55,50                               MINV 084
C          50 IK=NK+I                                     MINV 085
C          A(IK)=A(IK)/(-BIGA)                           MINV 086
C          55 CONTINUE                                   MINV 087
C
C          REDUCE MATRIX                                  MINV 088
C
C          DO 65 I=1,N                                   MINV 089
C          IK=NK+I                                       MINV 090
C          IJ=I-N                                         MINV 091
C          DO 65 J=1,N                                   MINV 092
C          IJ=IJ+N                                       MINV 093
C          IF(I-K) 60,65,60                               MINV 094
C          60 IF(J-K) 62,65,62                           MINV 095
C          62 KJ=IJ-I+K                                   MINV 096
C          A(IJ)=A(IK)+A(KJ)+A(IJ)                       MINV 097
C          65 CONTINUE                                   MINV 098
C
C          DIVIDE ROW BY PIVOT                            MINV 099
C
C          KJ=K-N                                         MINV 100
C          DO 75 J=1,N                                   MINV 101

```

```

      KJ=KJ+N
      IF(J-K) 70,75,70
70  A(KJ)=A(KJ)/BIGA
75  CONTINUE
C
      PRODUCT OF PIVOTS
C
      D=D*BIGA
C
      REPLACE PIVOT BY RECIPROCAL
C
      A(KK)=1.0/BIGA
90  CONTINUE
C
      FINAL ROW AND COLUMN INTERCHANGE
C
      K=N
      K=(K-1)
100 IF(K) 150,150,105
      IF(L(K))
105 IF(I-K) 120,120,108
108 JC=N+(K-1)
      JR=N+(I-1)
      DO 110 J=1,N
      JK=J+J
      HOLD=A(JK)
      JI=JR+J
      A(JK)=-A(JI)
110 A(JI)=HOLD
      J=M(K)
120 IF(J-K) 100,100,125
125 KI=K-N
      DO 130 I=1,N
      KI=KI+N
      HOLD=A(KI)
      JI=KI-K+J
      A(KI)=-A(JI)
130 A(JI)=HOLD
      GO TO 100
150 RETURN
      END
C
C .....
C
C SUBROUTINE ORDER
C
C PURPOSE
C CONSTRUCT FROM A LARGER MATRIX OF CORRELATION COEFFICIENTS
C A SUBSET MATRIX OF INTERCORRELATIONS AMONG INDEPENDENT
C VARIABLES AND A VECTOR OF INTERCORRELATIONS OF INDEPENDENT
C VARIABLES WITH DEPENDENT VARIABLE. THIS SUBROUTINE IS
C NORMALLY USED IN THE PERFORMANCE OF MULTIPLE AND POLYNOMIAL
C REGRESSION ANALYSES.
C
C USAGE
C CALL ORDER (M,R,NDEP,K,ISAVE,RX,RY)
C
C DESCRIPTION OF PARAMETERS
C M - NUMBER OF VARIABLES AND ORDER OF MATRIX R.
C R - INPUT MATRIX CONTAINING CORRELATION COEFFICIENTS.
C
      MINV 127
      MINV 128
      MINV 129
      MINV 130
      MINV 131
      MINV 132
      MINV 133
      MINV 134
      MINV 135
      MINV 136
      MINV 137
      MINV 138
      MINV 139
      MINV 140
      MINV 141
      MINV 142
      MINV 143
      MINV 144
      MINV 145
      MINV 146
      MINV 147
      MINV 148
      MINV 149
      MINV 150
      MINV 151
      MINV 152
      MINV 153
      MINV 154
      MINV 155
      MINV 156
      MINV 157
      MINV 158
      MINV 159
      MINV 160
      MINV 161
      MINV 162
      MINV 163
      MINV 164
      MINV 165
      MINV 166
      MINV 167
      ORDER001
      ORDER002
      ORDER003
      ORDER004
      ORDER005
      ORDER006
      ORDER007
      ORDER008
      ORDER009
      ORDER010
      ORDER011
      ORDER012
      ORDER013
      ORDER014
      ORDER015
      ORDER016
      ORDER017
      ORDER018
      ORDER019
C
C THIS SUBROUTINE EXPECTS ONLY UPPER TRIANGULAR
C PORTION OF THE SYMMETRIC MATRIX TO BE STORED (EY
C COLUMN) IN R. (STORAGE MODE OF 1)
C
C NDEP - THE SUBSCRIPT NUMBER OF THE DEPENDENT VARIABLE.
C K - NUMBER OF INDEPENDENT VARIABLES TO BE INCLUDED
C IN THE FORTHCOMING REGRESSION.
C
C ISAVE - INPUT VECTOR OF LENGTH K+1 CONTAINING, IN ASCENDING
C ORDER, THE SUBSCRIPT NUMBERS OF K INDEPENDENT
C VARIABLES TO BE INCLUDED IN THE FORTHCOMING REGRES-
C SION.
C
C UPON RETURNING TO THE CALLING ROUTINE, THIS VECTOR
C CONTAINS, IN ADDITION, THE SUBSCRIPT NUMBER OF
C THE DEPENDENT VARIABLE IN K+1 POSITION.
C
C RX - OUTPUT MATRIX (K X K) CONTAINING INTERCORRELATIONS
C AMONG INDEPENDENT VARIABLES TO BE USED IN FORTH-
C COMING REGRESSION.
C
C RY - OUTPUT VECTOR OF LENGTH K CONTAINING INTERCORRELAT-
C IONS OF INDEPENDENT VARIABLES WITH DEPENDENT
C VARIABLES.
C
C REMARKS
C NONE
C
C SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED
C NONE
C
C METHOD
C FROM THE SUBSCRIPT NUMBERS OF THE VARIABLES TO BE INCLUDED
C IN THE FORTHCOMING REGRESSION, THE SUBROUTINE CONSTRUCTS THE
C MATRIX RX AND THE VECTOR RY.
C
C .....
C
C SUBROUTINE ORDER (M,R,NDEP,K,ISAVE,RX,RY)
C DIMENSION R(11),ISAVE(1),RX(11),RY(1)
C
C .....
C
C IF A DOUBLE PRECISION VERSION OF THIS ROUTINE IS DESIRED, THE
C C IN COLUMN 1 SHOULD BE REMOVED FROM THE DOUBLE PRECISION
C STATEMENT WHICH FOLLOWS.
C
C DOUBLE PRECISION R,RX,RY
C
C THE C MUST ALSO BE REMOVED FROM DOUBLE PRECISION STATEMENTS
C APPEARING IN OTHER ROUTINES USED IN CONJUNCTION WITH THIS
C ROUTINE.
C
C .....
C
C COPY INTERCORRELATIONS OF INDEPENDENT VARIABLES
C WITH DEPENDENT VARIABLE
C
C MM=0
C DO 130 J=1,K
C L2=ISAVE(J)
C IF(NDEP-L2) 122, 123, 123
C 122 L=NDEP+(L2*L2-L2)/2
C GO TO 125
C 123 L=L2*(NDEP-NDEP)/2
C
      ORDER020
      ORDER021
      ORDER022
      ORDER023
      ORDER024
      ORDER025
      ORDER026
      ORDER027
      ORDER028
      ORDER029
      ORDER030
      ORDER031
      ORDER032
      ORDER033
      ORDER034
      ORDER035
      ORDER036
      ORDER037
      ORDER038
      ORDER039
      ORDER040
      ORDER041
      ORDER042
      ORDER043
      ORDER044
      ORDER045
      ORDER046
      ORDER047
      ORDER048
      ORDER049
      ORDER050
      ORDER051
      ORDER052
      ORDER053
      ORDER054
      ORDER055
      ORDER056
      ORDER057
      ORDER058
      ORDER059
      ORDER060
      ORDER061
      ORDER062
      ORDER063
      ORDER064
      ORDER065
      ORDER066
      ORDER067
      ORDER068
      ORDER069
      ORDER070
      ORDER071
      ORDER072
      ORDER073
      ORDER074
      ORDER075
      ORDER076
      ORDER077
      ORDER078
      ORDER079

```

```

125 RY(IJ)=R(L)
C
C COPY A SUBSET MATRIX OF INTERCORRELATIONS AMONG
C INDEPENDENT VARIABLES
C
DO 130 I=1,K
L1=ISAVE(I)
IF(L1-L2) 127, 128, 128
127 L=L1+(L2*L2-L2)/2
GO TO 129
128 L=L2+(L1*L1-L1)/2
129 MM=MM+1
130 RX(MM)=R(L)
C
C PLACE THE SUBSCRIPT NUMBER OF THE DEPENDENT
C VARIABLE IN ISAVE(K+1)
C
ISAVE(K+1)=NDEP
RETURN
END
C
C .....
C
C SUBROUTINE GDATA
C
C PURPOSE
C GENERATE INDEPENDENT VARIABLES UP TO THE M-TH POWER (THE
C HIGHEST DEGREE POLYNOMIAL SPECIFIED) AND COMPUTE MEANS,
C STANDARD DEVIATIONS, AND CORRELATION COEFFICIENTS. THIS
C SUBROUTINE IS NORMALLY CALLED BEFORE SUBROUTINES ORDER,
C MINV AND MULTR IN THE PERFORMANCE OF A POLYNOMIAL
C REGRESSION.
C
C USAGE
C CALL GDATA (N,M,X,XBAR,STD,D,SUMSQ)
C
C DESCRIPTION OF PARAMETERS
C N - NUMBER OF OBSERVATIONS.
C M - THE HIGHEST DEGREE POLYNOMIAL TO BE FITTED.
C X - INPUT MATRIX (N BY M+1). WHEN THE SUBROUTINE IS
C CALLED, DATA FOR THE INDEPENDENT VARIABLE ARE
C STORED IN THE FIRST COLUMN OF MATRIX X, AND DATA FOR
C THE DEPENDENT VARIABLE ARE STORED IN THE LAST
C COLUMN OF THE MATRIX. UPON RETURNING TO THE
C CALLING ROUTINE, GENERATED POWERS OF THE INDEPENDENT
C VARIABLE ARE STORED IN COLUMNS 2 THROUGH M.
C XBAR - OUTPUT VECTOR OF LENGTH M+1 CONTAINING MEANS OF
C INDEPENDENT AND DEPENDENT VARIABLES.
C STD - OUTPUT VECTOR OF LENGTH M+1 CONTAINING STANDARD
C DEVIATIONS OF INDEPENDENT AND DEPENDENT VARIABLES.
C D - OUTPUT MATRIX (ONLY UPPER TRIANGULAR PORTION OF THE
C SYMMETRIC MATRIX OF M+1 BY M+1) CONTAINING CORRELA-
C TION COEFFICIENTS. (STORAGE MODE OF 1)
C SUMSQ - OUTPUT VECTOR OF LENGTH M+1 CONTAINING SUMS OF
C PRODUCTS OF DEVIATIONS FROM MEANS OF INDEPENDENT
C AND DEPENDENT VARIABLES.
C
C REMARKS
C N MUST BE GREATER THAN M+1.
C IF M IS EQUAL TO 5 OR GREATER, SINGLE PRECISION MAY NOT BE

```

```

GRDER080
GRDER081
GRDER082
GRDER083
GRDER084
GRDER085
GRDER086
GRDER087
GRDER088
GRDER089
GRDER090
GRDER091
GRDER092
GRDER093
GRDER094
GRDER095
GRDER096
GRDER097
GRDER098
GRDER099
GDATA001
GDATA002
GDATA003
GDATA004
GDATA005
GDATA006
GDATA007
GDATA008
GDATA009
GDATA010
GDATA011
GDATA012
GDATA013
GDATA014
GDATA015
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GDATA021
GDATA022
GDATA023
GDATA024
GDATA025
GDATA026
GDATA027
GDATA028
GDATA029
GDATA030
GDATA031
GDATA032
GDATA033
GDATA034
GDATA035
GDATA036
GDATA037
GDATA038
GDATA039
GDATA040

```

```

C SUFFICIENT TO GIVE SATISFACTORY COMPUTATIONAL RESULTS.
C
C SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED
C NONE
C
C METHOD
C REFER TO B. OSTLE, 'STATISTICS IN RESEARCH', THE IOWA STATE
C COLLEGE PRESS, 1954, CHAPTER 6.
C
C .....
C SUBROUTINE GDATA (N,M,X,XBAR,STD,D,SUMSQ)
C DIMENSION X(1),XBAR(1),STD(1),D(1),SUMSQ(1)
C
C .....
C IF A DOUBLE PRECISION VERSION OF THIS ROUTINE IS DESIRED, THE
C C IN COLUMN 1 SHOULD BE REMOVED FROM THE DOUBLE PRECISION
C STATEMENT WHICH FOLLOWS.
C
C DOUBLE PRECISION X,XBAR,STD,D,SUMSQ,T1,T2,DABS,DSQRT
C
C THE C MUST ALSO BE REMOVED FROM DOUBLE PRECISION STATEMENTS
C APPEARING IN OTHER ROUTINES USED IN CONJUNCTION WITH THIS
C ROUTINE.
C
C THE DOUBLE PRECISION VERSION OF THIS SUBROUTINE MUST ALSO
C CONTAIN DOUBLE PRECISION FORTRAN FUNCTIONS. SQRT AND ABS IN
C STATEMENT 180 MUST BE CHANGED TO DSQRT AND DABS.
C
C .....
C GENERATE INDEPENDENT VARIABLES
C
C IF(M-1) 105, 105, 90
C L1=0
C DO 100 I=2,M
C L1=L1+M
C DO 100 J=1,N
C L=L1+J
C K=L-M
C 100 X(L)=X(K)*X(I)
C
C CALCULATE MEANS
C
C 105 MM=M+1
C DF=N
C L=0
C DO 115 I=1,MM
C XBAR(I)=0.0
C DO 110 J=1,N
C L=L+1
C 110 XBAR(I)=XBAR(I)+X(L)
C 115 XBAR(I)=XBAR(I)/DF
C
C DO 130 I=1,MM
C 130 STD(I)=0.0
C
C CALCULATE SUMS OF CROSS-PRODUCTS OF DEVIATIONS
C
C L=((MM+1)*MM)/2

```

```

GDATA041
GDATA042
GDATA043
GDATA044
GDATA045
GDATA046
GDATA047
GDATA048
GDATA049
GDATA050
GDATA051
GDATA052
GDATA053
GDATA054
GDATA055
GDATA056
GDATA057
GDATA058
GDATA059
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GDATA091
GDATA092
GDATA093
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GDATA095
GDATA096
GDATA097
GDATA098
GDATA099
GDATA100

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```

150 DD 150 I=1,L
O(I)=0.0
DD 170 K=1,M
L=0
DD 170 J=1,MM
L2=N*(J-1)+K
T2=X(L2)-XBAR(J)
STD(J)=STD(I)+T2
DD 170 I=1,J
L1=N*(I-1)+K
T1=X(L1)-XBAR(I)
L=L+1
170 D(L)=D(L)+T1+T2
L=0
DD 175 J=1,MM
DD 175 I=1,J
L=L+1
175 D(L)=D(L)-STD(I)*STD(J)/DF
L=0
DD 180 I=1,MM
L=L+1
SUMSQ(I)=D(L)
180 STD(I)=CSQRT(DABS(D(L)))
C
C CALCULATE CORRELATION COEFFICIENTS
C
L=0
DD 190 J=1,MM
DD 190 I=1,J
L=L+1
190 D(L)=D(L)/(STD(I)*STD(J))
C
C CALCULATE STANDARD DEVIATIONS
C
DF=SQRT(DF-1.0)
DD 200 I=1,MM
200 STD(I)=STD(I)/DF
RETURN
END
C
C .....
C SUBROUTINE MULTR
C
C PURPOSE
C PERFORM A MULTIPLE LINEAR REGRESSION ANALYSIS FOR A DEPENDENT VARIABLE AND A SET OF INDEPENDENT VARIABLES. THIS SUBROUTINE IS NORMALLY USED IN THE PERFORMANCE OF MULTIPLE AND POLYNOMIAL REGRESSION ANALYSES.
C
C USAGE
C CALL MULTR (N,K,XBAR,STD,D,RX,RY,ISAVE,B,SB,T,ANS)
C
C DESCRIPTION OF PARAMETERS
C N - NUMBER OF OBSERVATIONS.
C K - NUMBER OF INDEPENDENT VARIABLES IN THIS REGRESSION.
C XBAR - INPUT VECTOR OF LENGTH M CONTAINING MEANS OF ALL VARIABLES. M IS NUMBER OF VARIABLES IN OBSERVATIONS.
C STD - INPUT VECTOR OF LENGTH M CONTAINING STANDARD DEVIATIONS OF ALL VARIABLES.

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GDATA101
GDATA102
GDATA103
GDATA104
GDATA105
GDATA106
GDATA107
GDATA108
GDATA109
GDATA110
GDATA111
GDATA112
GDATA113
GDATA114
GDATA115
GDATA116
GDATA117
GDATA118
GDATA119
GDATA120
GDATA121
GDATA122
GDATA123
GDATA124
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GDATA128
GDATA129
GDATA130
GDATA131
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GDATA133
GDATA134
GDATA135
GDATA136
GDATA137
GDATA138
GDATA139
MULTR001
MULTR002
MULTR003
MULTR004
MULTR005
MULTR006
MULTR007
MULTR008
MULTR009
MULTR010
MULTR011
MULTR012
MULTR013
MULTR014
MULTR015
MULTR016
MULTR017
MULTR018
MULTR019
MULTR020
MULTR021

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C
C - INPUT VECTOR OF LENGTH M CONTAINING THE DIAGONAL OF THE MATRIX OF SUMS OF CROSS-PRODUCTS OF DEVIATIONS FROM MEANS FOR ALL VARIABLES.
C
C RX - INPUT MATRIX (K X K) CONTAINING THE INVERSE OF INTERCORRELATIONS AMONG INDEPENDENT VARIABLES.
C
C RY - INPUT VECTOR OF LENGTH K CONTAINING INTERCORRELATIONS OF INDEPENDENT VARIABLES WITH DEPENDENT VARIABLE.
C
C ISAVE - INPUT VECTOR OF LENGTH K+1 CONTAINING SUBSCRIPTS OF INDEPENDENT VARIABLES IN ASCENDING ORDER. THE SUBSCRIPT OF THE DEPENDENT VARIABLE IS STORED IN THE LAST, K+1, POSITION.
C
C B - OUTPUT VECTOR OF LENGTH K CONTAINING REGRESSION COEFFICIENTS.
C
C SB - OUTPUT VECTOR OF LENGTH K CONTAINING STANDARD DEVIATIONS OF REGRESSION COEFFICIENTS.
C
C T - OUTPUT VECTOR OF LENGTH K CONTAINING T-VALUES.
C
C ANS - OUTPUT VECTOR OF LENGTH 10 CONTAINING THE FOLLOWING INFORMATION..
C ANS(1) INTERCEPT
C ANS(2) MULTIPLE CORRELATION COEFFICIENT
C ANS(3) STANDARD ERROR OF ESTIMATE
C ANS(4) SUM OF SQUARES ATTRIBUTABLE TO REGRESSION (SSAR)
C ANS(5) DEGREES OF FREEDOM ASSOCIATED WITH SSAR
C ANS(6) MEAN SQUARE OF SSAR
C ANS(7) SUM OF SQUARES OF DEVIATIONS FROM REGRESSION (SSDR)
C ANS(8) DEGREES OF FREEDOM ASSOCIATED WITH SSDR
C ANS(9) MEAN SQUARE OF SSDR
C ANS(10) F-VALUE
C
REMARKS
N MUST BE GREATER THAN K+1.
C
SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED
NONE
C
METHOD
THE GAUSS-JORDAN METHOD IS USED IN THE SOLUTION OF THE NORMAL EQUATIONS. REFER TO W. W. COOLEY AND P. R. LOHNE, 'MULTIVARIATE PROCEDURES FOR THE BEHAVIORAL SCIENCES', JOHN WILEY AND SONS, 1962, CHAPTER 3, AND B. OSTLE, 'STATISTICS IN RESEARCH', THE IDNA STATE COLLEGE PRESS, 1964, CHAPTER 8.
C
C .....
C SUBROUTINE MULTR (N,K,XBAR,STD,D,RX,RY,ISAVE,B,SB,T,ANS)
C DIMENSION XBAR(1),STD(1),D(1),RX(1),RY(1),ISAVE(1),B(1),SB(1),T(1),ANS(1)
C
C .....
C IF A DOUBLE PRECISION VERSION OF THIS ROUTINE IS DESIRED, THE C IN COLUMN 1 SHOULD BE REMOVED FROM THE DOUBLE PRECISION STATEMENT WHICH FOLLOWS.
C
DOUBLE PRECISION XBAR,STD,D,RX,RY,B,SB,T,ANS,FM,BO,SSAR,SSDR,SY,
FN,FK,SSARM,SSDRM,F,DABS,DSQRT

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MULTR022
MULTR023
MULTR024
MULTR025
MULTR026
MULTR027
MULTR028
MULTR029
MULTR030
MULTR031
MULTR032
MULTR033
MULTR034
MULTR035
MULTR036
MULTR037
MULTR038
MULTR039
MULTR040
MULTR041
MULTR042
MULTR043
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MULTR065
MULTR066
MULTR067
MULTR068
MULTR069
MULTR070
MULTR071
MULTR072
MULTR073
MULTR074
MULTR075
MULTR076
MULTR077
MULTR078
MULTR079
MULTR080
MULTR081

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C		MULTR082	DO 130 J=1,K	MULTR142
C	THE C MUST ALSO BE REMOVED FROM DOUBLE PRECISION STATEMENTS	MULTR083	LI=K*(J-1)+J	MULTR143
C	APPEARING IN OTHER ROUTINES USED IN CONJUNCTION WITH THIS	MULTR084	L=ISAVE(J)	MULTR144
C	ROUTINE.	MULTR085	125 SBI(J)=DSQRT(DABS((RY(L1))/D(L1)*SY))	MULTR145
C		MULTR086		MULTR146
C	THE DOUBLE PRECISION VERSION OF THIS SUBROUTINE MUST ALSO	MULTR087	C	MULTR147
C	CONTAIN DOUBLE PRECISION FORTRAN FUNCTIONS. SQRT AND ABS IN	MULTR088	C	MULTR148
C	STATEMENTS 122, 125, AND 135 MUST BE CHANGED TO DSQRT AND DABS.	MULTR089	C	MULTR149
C	.....	MULTR090	C	MULTR150
C		MULTR091	C	MULTR151
C		MULTR092	C	MULTR152
C	MM=K+1	MULTR093	C	MULTR153
C		MULTR094	C	MULTR154
C	BETA WEIGHTS	MULTR095	C	MULTR155
C		MULTR096	C	MULTR156
C	DO 100 J=1,K	MULTR097	C	MULTR157
C	3(J)=0.0	MULTR098	C	MULTR158
C	DO 110 J=1,K	MULTR099	C	MULTR159
C	LI=K*(J-1)	MULTR100	C	MULTR160
C	DO 110 I=1,K	MULTR101	C	MULTR161
C	L=LI+I	MULTR102	C	MULTR162
C	110 B(J)=B(J)+RY(I)*RX(L)	MULTR103	C	MULTR163
C	RW=0.0	MULTR104	C	MULTR164
C	BO=0.0	MULTR105	C	MULTR165
C	L1=ISAVE(MM)	MULTR106	C	MULTR166
C		MULTR107	C	MULTR167
C	COEFFICIENT OF DETERMINATION	MULTR108	C	MULTR168
C		MULTR109	C	MULTR169
C	DO 120 I=1,K	MULTR110	C	MULTR170
C	RM=RM+B(I)*RY(I)	MULTR111	C	MULTR171
C		MULTR112	C	MULTR172
C	REGRESSION COEFFICIENTS	MULTR113	C	MULTR173
C		MULTR114		
C	L=ISAVE(I)	MULTR115	ANS(1)=BO	
C	B(I)=B(I)*(STD(L1)/STD(L))	MULTR116	ANS(2)=PM	
C		MULTR117	ANS(3)=SY	
C	INTERCEPT	MULTR118	ANS(4)=SSAR	
C		MULTR119	ANS(5)=FK	
C	120 BO=BO+B(I)*XBAR(L)	MULTR120	ANS(6)=SSAR*	
C	BO=XBAR(L1)-BO	MULTR121	ANS(7)=SSDR	
C		MULTR122	ANS(8)=FN	
C	SUM OF SQUARES ATTRIBUTABLE TO REGRESSION	MULTR123	ANS(9)=SSDR*	
C		MULTR124	ANS(10)=F	
C	SSAR=RM*D(L1)	MULTR125	RETURN	
C		MULTR126	END	
C	MULTIPLE CORRELATION COEFFICIENT	MULTR127		
C		MULTR128		
C	122 RM=DSQRT(DABS(RM))	MULTR129		
C		MULTR130		
C	SUM OF SQUARES OF DEVIATIONS FROM REGRESSION	MULTR131		
C		MULTR132		
C	SSDR=D(L1)-SSAR	MULTR133		
C		MULTR134		
C	VARIANCE OF ESTIMATE	MULTR135		
C		MULTR136		
C	FN=N-K-1	MULTR137		
C	SY=SSDR/FN	MULTR138		
C		MULTR139		
C	STANDARD DEVIATIONS OF REGRESSION COEFFICIENTS	MULTR140		
C		MULTR141		

POLYNOMIAL REGRESSION.....A1

NUMBER OF OBSERVATIONS 10

STARTING TEMPERATURE = 715.00

TEMPERATURE DIFFERENCE = 9

TEMPERATURE INCREMENT = 1.00

POLYNOMIAL REGRESSION OF DEGREE 1

INTERCEPT 0.2242837785164737D 01

REGRESSION COEFFICIENTS  
-0.1927565275430233D-02

ANALYSIS OF VARIANCE FOR 1 DEGREE POLYNOMIAL

SOURCE OF VARIATION	DEGREE OF FREEDOM	SUM OF SQUARES	MEAN SQUARE	F VALUE	IMPROVEMENT IN TERMS OF SUM OF SQUARES
DUE TO REGRESSION	1	0.00031	0.00031	421.25456	0.00031
DEVIATION ABOUT REGRESSION	8	0.00001	0.00000		
TOTAL	9	0.00031			

POLYNOMIAL REGRESSION OF DEGREE 2

INTERCEPT -0.1427062306580634D 03

REGRESSION COEFFICIENTS  
0.2439232085470450D 00 -0.1042473832536649D-03

ANALYSIS OF VARIANCE FOR 2 DEGREE POLYNOMIAL

SOURCE OF VARIATION	DEGREE OF FREEDOM	SUM OF SQUARES	MEAN SQUARE	F VALUE	IMPROVEMENT IN TERMS OF SUM OF SQUARES
DUE TO REGRESSION	2	0.00031	0.00016	13133.43067	0.00001
DEVIATION ABOUT REGRESSION	7	0.00000	0.00000		
TOTAL	9	0.00031			

POLYNOMIAL REGRESSION OF DEGREE 3

NO IMPROVEMENT

POLYNOMIAL REGRESSION.....A1

POLYNOMIAL REGRESSION OF DEGREE 2

TABLE OF RESIDUALS

OBSERVATION NO.	X VALUE	Y VALUE	Y ESTIMATE	RESIDUAL	ERROR
1	1174.66900	-0.02255	-0.02267	0.00012	-0.54195
2	1175.66900	-0.02379	-0.02376	-0.00004	0.15022
3	1176.66900	-0.02516	-0.02506	-0.00011	0.42278
4	1177.66900	-0.02667	-0.02658	-0.00009	0.34602
5	1178.66900	-0.02834	-0.02829	-0.00005	0.15947
6	1179.66900	-0.03019	-0.03022	0.00002	-0.08216
7	1180.66900	-0.03226	-0.03235	0.00009	-0.27111
8	1181.66900	-0.03458	-0.03470	0.00012	-0.35321
9	1182.66900	-0.03720	-0.03725	0.00006	-0.15144
10	1183.66900	-0.04015	-0.04001	-0.00014	0.35469

```

C ..... SYSS 10
C ..... SYSS 20
C ..... SYSS 30
SUBROUTINE SYSSOL(M,N,A,B,ALPHA,A1,B1,TOL,DELEPS,Y,Y,
* ABT,FF,JACOB,JERROR)
*
* IMPLICIT REAL*(8(A-H,O-Z))
* EXTERNAL JACOB
* DIMENSION X(1)
* DIMENSION ALPHA(10),A1(10,10),B1(10,10),Y(2600),ABT(10)
* DIMENSION F(2600),LU(2600),RES(2600),SK(2600),
* ABSEX(10),ABR(10),TEMP(10)
* DIMENSION AA(50),BB(50),CC(20)
COMMON /CI/ EPSNU
LOGICAL DIVNEW
C .....
C*
C* PURPOSE : THIS IS A VARIABLE ORDER, VARIABLE
C* (UNIFORM) STEP SOLVER FOR TWO-POINT BOUNDARY VALUE FIRST-
C* ORDER SMOOTH NONLINEAR SYSTEMS OF THE FORM
C*
C* (1)  $Y' = F(X,Y)$  ;  $A1 \cdot Y(A) + B1 \cdot Y(B) = ALPHA$ 
C*
C* IT ATTEMPTS TO PRODUCE A DISCRETE SOLUTION WITH MAXIMUM
C* ABSOLUTE ERROR LESS THAN TOL ON A UNIFORM GRID CONTAIN-
C* ING THE ONE DEFINED BY THE USER WITH THE PARAMETER N.
C* THE METHOD, MORE DETAILS AND TESTS ARE DESCRIBED IN
C* "A VARIABLE ORDER, VARIABLE STEP, FINITE DIFFERENCE
C* METHOD FOR NONLINEAR MULTIPPOINT BOUNDARY VALUE PROBLEMS"
C* BY M. LENTINI AND V. PEREYRA, PUB. 73-06, DEPTO. DE
C* COMPUTACION, FAC. DE CIENCIAS, UNIVERSIDAD CENTRAL DE
C* VENEZUELA (1973).
C*
C*
C .....
CONTINUE
C .....
C*
C* USE : PRESENT DIMENSIONING OF ARRAYS ALLOWS PROCESSING
C* OF SYSTEMS OF UP TO
C*  $M \times N$  EQUATIONS
C* ON GRIDS WITH UP TO
C*  $M \times N$  POINTS.
C* THESE SIZES CAN BE VARIED BY CHANGING THE
C* DIMENSIONS APPROPRIATELY.
C*
C* THE USER MUST SPECIFY EACH OF THE FOLLOWINGS:
C*
C* M - NUMBER OF EQUATIONS
C* N - NUMBER OF POINTS IN THE BASIC GRID (COUNTING THE
C* END POINTS). N MUST BE > 3.
C* UPON EXIT, N WILL CONTAIN THE FINAL GRID SIZE
C* IN WHICH Y HAS BEEN COMPUTED.
C* A - LEFT BOUNDARY POINT
C* B - RIGHT BOUNDARY POINT
C* ALPHA A1 , B1 - ARRAYS OF DIMENSIONS MMAX
C* CONTAINING BOUNDARY CONDITIONS.SEE(1)
C*
C* TOL - DESIRED FINAL ABSOLUTE ERROR.
C*
C* DELEPS - (A) IF DELEPS <= 0 THEN THE PROGRAM SETS Y=0 AS
C* INITIAL VALUES.

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C* IF DELEPS IS SET TO ANY VALUE >= 1 THEN THE
C* PROGRAM ASSUMES THAT THE USER WILL GIVE
C* INITIAL VALUES FOR Y. THESE INITIAL VALUES
C* (ON THE INITIAL MESH) MUST BE STORED AS
C* INDICATED BELOW (SEE OUTPUT PARAMETERS).
C* (B) A CONTINUATION METHOD CAN BE EMPLOYED AS AN
C* OPTION IN SEVERELY NON LINEAR CASES IN
C* ORDER TO GENERATE GOOD INITIAL VALUES. THIS
C* PRESUPPOSES THAT THE USER HAS EMBEDDED HIS
C* PROBLEM IN A ONE PARAMETER FAMILY:
C*  $Y' = F(X, Y, EPSNU)$ 
C* SUCH THAT FOR EPSNU = 0 THE RESULTING
C* PROBLEM IS "SIMPLER", AND FOR EPSNU = 1
C* THE ORIGINAL PROBLEM IS RECOVERED. THE
C* PROGRAM WILL AUTOMATICALLY ATTEMPT TO GO
C* FROM EPSNU = 0 TO EPSNU = 1 IN STEPS OF
C* DELEPS. THUS IN THIS CASE
C*  $0 < DELEPS < 1$ 
C* ONCE EPSNU >= 1 THE REGULAR PROCEDURE
C* CONTINUES. THE USER MUST GIVE INITIAL
C* VALUES FOR Y AS IN (A).
C*
C* WHEN USING THIS OPTION THE USER MUST
C* INCLUDE IN HIS SUBROUTINES FF, JACOB
C* THE COMMON CARDS
C* COMMON /CI/ EPSNU
C* IN ORDER TO DESCRIBE THE PARAMETRIZED
C* PROBLEM
C .....
CONTINUE
C .....
C*
C* *** USER PROVIDED SUBROUTINES ***
C*
C* FF - NAME OF SUBROUTINE FOR EVALUATING F(X,Y) ON
C* THE MESH. THE SUBROUTINE ITSELF MUST BE
C* PROVIDED BY THE USER AND IT SHOULD HAVE THE
C* FOLLOWING HEADING
C*
C* SUBROUTINE FF(X,Y,N,F)
C*
C* WHERE F IS THE VECTOR CONTAINING THE VALUES
C* OF F(X,Y). STORAGE OF F IS THE SAME AS THAT
C* OF Y (SEE BELOW).
C*
C* JACOB - NAME OF A SUBROUTINE FOR EVALUATING THE
C* JACOBIAN OF F(X,Y) AT A GIVEN GRID POINT.
C* THE SUBROUTINE ITSELF MUST BE PROVIDED BY
C* THE USER, AND IT SHOULD HAVE THE FOLLOWING
C* HEADING
C*
C* SUBROUTINE JACOB(X,Y,XJAC)
C*
C* WHERE XX IS THE GIVEN GRID POINT, YY IS
C* THE CURRENT VALUE OF Y AT XX, AND THE
C* MMAX*MMAX ARRAY XJAC SHOULD BE FILLED BY
C* THE USER WITH THE CORRESPONDING PARTIAL

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SYSS 600
SYSS 610
SYSS 620
SYSS 630
SYSS 640
SYSS 650
SYSS 660
SYSS 670
SYSS 680
SYSS 690
SYSS 700
SYSS 710
SYSS 720
SYSS 730
SYSS 740
SYSS 750
SYSS 760
SYSS 770
SYSS 780
SYSS 790
SYSS 800
SYSS 810
SYSS 820
SYSS 830
SYSS 840
SYSS 850
SYSS 860
SYSS 870
SYSS 880
SYSS 890
SYSS 900
SYSS 910
SYSS 920
SYSS 930
SYSS 940
SYSS 950
SYSS 960
SYSS 970
SYSS 980
SYSS 990
SYSS1000
SYSS1010
SYSS1020
SYSS1030
SYSS1040
SYSS1050
SYSS1060
SYSS1070
SYSS1080
SYSS1090
SYSS1100
SYSS1110
SYSS1120
SYSS1130
SYSS1140
SYSS1150
SYSS1160
SYSS1170
SYSS1180
SYSS1190

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C*      DERIVATIVES ACCORDING TO      *
C*      XJAC(I,J) = DFI/DYJ          *
C*      *                               *
C*.....*                               *
C*      CONTINUE                      *
C*.....*                               *
C*      *** OUTPUT PARAMETERS ***     *
C*      *                               *
C*      SEE M ABOVE                   *
C*      *                               *
C*      Y - NMAX*MMAX-VECTOR CONTAINING THE COMPUTED *
C*      SOLUTION ON THE FINAL GRID *
C*      THE GRID VECTOR FUNCTION Y(X) IS STORED *
C*      SEQUENTIALLY IN THE FORM: *
C*      Y1(X1),Y2(X1),...,YM(X1),Y1(X2),...,YM(X2), *
C*      Y THE PROGRAM SETS Y TO ZERO INITIALLY WHEN *
C*      DELEPS <= 0. *
C*      X - NMAX-VECTOR CONTAINING THE FINAL GRID POINTS. *
C*      ABT - M-VECTOR CONTAINING MAX ABSOLUTE ERROR ON THE *
C*      GRID FOR EACH COMPONENT OF THE APPROXIMATE *
C*      SOLUTION *
C*      *                               *
C*      JERROR - ERROR CODED INTEGER VARIABLE. *
C*      JERROR = 0 REQUIRED TOLERANCE WAS REACHED *
C*      JERROR = 1 M OR INITIAL N ARE OUT OF RANGE. *
C*      CORRECTING ACTION: CHECK THAT INPUT DATA *
C*      SATISFIES 0< M <= MMAX AND 3 < N <= NMAX *
C*      AND RERUN. *
C*      JERROR = 2 THE PROGRAM HAS ATTEMPTED TO USE A GRID *
C*      WITH MORE THAN NMAX POINTS *
C*      CORRECTING ACTION: IF PARTIAL RESULTS SEEM *
C*      REASONABLE, INCREASE MMAX*NMAX = 2600 AND *
C*      CHANGE DIMENSIONS IN ALL SUBROUTINES ACCORDING- *
C*      LY IN ORDER TO ALLOW FINER GRIDS *
C*.....*                               *
C*      CONTINUE                      *
C*.....*                               *
C*      *** OTHER SUBROUTINES NEEDED BY SYSSOL *** *
C*      *                               *
C*      SYSLIN - SOLVES A BLOCK LINEAR SYSTEM OF SPECIAL TYPE. *
C*      DGELG - SOLVES A LINEAR SYSTEM OF EQUATIONS *
C*      DARRAY - TRANSFORMS BETWEEN TWO AND ONE DIMENSIONAL *
C*      STORAGE. *
C*      U2DCGS - DEFERRED CORRECTOIN GENERATOR AND INTERPOLATION *
C*      CDEGEN - WEIGHT GENERATOR *
C*.....*                               *
C*.....*                               *
C*      IN CALLING PROGRAM THE PARAMETER ARRAY X MUST BE *
C*      DIMENSIONED AS XI(2600/M) *
C*      *                               *
C*      DIMENSIONS INVOLVED IN THE FOLLOWING PARAMETER ARRAYS *
C*      ARE MMAX = 10 AND NMAX*MMAX = 2600, AND THEY MUST BE *
C*      DIMENSIONED ACCORDINGLY IN THE CALLING PROGRAM *
C*      *                               *
C*      DIMENSION ALPHA(10),A1(10,10),B1(10,10),Y(2600),ABT(10) *

```

```

SYSS1200
SYSS1210
SYSS1220
SYSS1230
SYSS1240
SYSS1250
SYSS1260
SYSS1270
SYSS1280
SYSS1290
SYSS1300
SYSS1310
SYSS1320
SYSS1330
SYSS1340
SYSS1350
SYSS1360
SYSS1370
SYSS1380
SYSS1390
SYSS1400
SYSS1410
SYSS1420
SYSS1430
SYSS1440
SYSS1450
SYSS1460
SYSS1470
SYSS1480
SYSS1490
SYSS1500
SYSS1510
SYSS1520
SYSS1530
SYSS1540
SYSS1550
SYSS1560
SYSS1570
SYSS1580
SYSS1590
SYSS1600
SYSS1610
SYSS1620
SYSS1630
SYSS1640
SYSS1650
SYSS1660
SYSS1670
SYSS1680
SYSS1690
SYSS1700
SYSS1710
SYSS1720
SYSS1730
SYSS1740
SYSS1750
SYSS1760
SYSS1770
SYSS1780
SYSS1790

```

```

C*      *                               *
C*      FOLLOWING ARRAYS ARE WORKING AREAS. *
C*      *                               *
C*      DIMENSION F(2600),UU(2600),SK(2600), *
C*      APSEX(10),APR(10),TEMP(10) *
C*      *                               *
C*      WORKING AREAS WITH SIZES RELATED TO MAX. NUMBER OF *
C*      DEFERRED CORRECTIONS = 20 , WHICH SHOULD BE ADEQUATED *
C*      FOR ALL PURPOSES. *
C*.....*                               *
C*      *                               *
C*      MMAX=10 *
C*      NMAX=2600/M *
C*      IF (M.LE.0).OR.(M.GT.NMAX).OR.(N.LE.3).OR.(N.GT.NMAX)) GO TO 1 *
C*      GO TO 10 *
C*      1 JERRCR=1 *
C*      RETURN *
C*.....*                               *
C<<<<...      ERROR EXIT 1      *
C*      *                               *
C*      INITIALIZATION *
C*      *                               *
C*      10 MPN=M*N *
C*      *                               *
C*      THIS CONSTANT IS MACHINE DEPENDENT: *
C*      EPSMAC IS APPROXIMATELY 10* RELATIVE PRECISION OF *
C*      FLOATING POINT ARITHMETIC *
C*.....*                               *
C*      *                               *
C*      EPSMAC=5.0-15 *
C*      EPSNU=0.00 *
C*      JERPD=0 *
C*      M1=M+1 *
C*      M2=M*M *
C*      IF (DELEPS.GT.0.00) GO TO 60 *
C*      *                               *
C*      FIRST APPROXIMATION FOR Y *
C*.....*                               *
C*      *                               *
C*      40 DO 50 I=1,MPN *
C*      50 Y(I)=0.00 *
C*      *                               *
C*      60 B(1)=1.00 *
C*      DO 105 I=2,50 *
C*      105 B(I)=0.00 *
C*.....*                               *
C*      *                               *
C*      IN CORRECTION NU=0 THE RESIDUAL IN THE NEWTON ITERATION *
C*      MUST BE REDUCED IN FORM BELOW EPS=C*M**4 , WHERE C IS *
C*      A SMALL CONSTANT. FOR NU > 0, WE USE THE ERROR ESTIMATE *
C*      CORRESPONDING TO CORRECTION (NU-1) ; ERROLD, IN ORDER TO *
C*      OBTAIN THE NEW EPS= C*M**2*ERROLD. IN ALL CASES EPS *
C*      IS NOT ALLOWED TO BE SMALLER THAN EPSMAC. *
C*.....*                               *
C*      *                               *
C*      EPS=DMAX1(EPSMAC,.01D0*(8-A)/(N-1))**4 *
C*      NU=0 *

```

```

SYSS1800
SYSS1810
SYSS1820
SYSS1830
SYSS1840
SYSS1850
SYSS1860
SYSS1870
SYSS1880
SYSS1890
SYSS1900
SYSS1910
SYSS1920
SYSS1930
SYSS1940
SYSS1950
SYSS1960
SYSS1970
SYSS1980
SYSS1990
SYSS2000
SYSS2010
SYSS2020
SYSS2030
SYSS2040
SYSS2050
SYSS2060
SYSS2070
SYSS2080
SYSS2090
SYSS2100
SYSS2110
SYSS2120
SYSS2130
SYSS2140
SYSS2150
SYSS2160
SYSS2170
SYSS2180
SYSS2190
SYSS2200
SYSS2210
SYSS2220
SYSS2230
SYSS2240
SYSS2250
SYSS2260
SYSS2270
SYSS2280
SYSS2290
SYSS2300
SYSS2310
SYSS2320
SYSS2330
SYSS2340
SYSS2350
SYSS2360
SYSS2370
SYSS2380
SYSS2390

```

```

C.....>>>      ENTER UPON STEP HALVING      .....
C
120  ERROLD=1.0D20
    KMAX=(N-2)/2
    MPNM=MPN-M
    C1=0.8D0
    N1=N-1
    H=(B-A)/M1
    HCUA=H**2
    DO 150 I=1,MPNM
150  SK(I)=0.D0
    X(I)=A
    X(N)=B
    DO 200 I=2,N1
    I1=I-1
200  X(I)=A+I1*H
    IF(NU .EQ. 0) GO TO 405
C
C      WHEN STEP-HALVING WE HAVE TO INITIALIZE SK IF NU .GT. 0
C
    CALL FFI(X,Y,N,F)
    CALL U2DCGS(NU,2,2,N1,M,AA,F,RES,IEPROR)
    DO 300 I=1,MPNM
300  SK(I)=H*RES(I)
C
C.....>>>      NEWTON ITERATION STARTS      .....
C
405  INWT=0
    REOLD=1.0D20
    DIVNEW=.FALSE.
    IF(NU+1.LE.KMAX) GO TO 410
C
C      MAXIMUM NUMBER OF CORRECTIONS ON THIS MESH HAS BEEN
C      REACHED. GO TO 'STEP HALVING'.
C
    NU=NU+1
    GO TO 2600
C.....>>>      LABEL 410 IS INPUT FOR NEWTON ITERATION      .....
C
C      RESIDUAL COMPUTATION
C
410  RABS=0.D0
    DO 700 I=1,M
    SUM=ALPHA(I)
    DO 600 J=1,M
600  SUM=SUM-A1(I,J)*Y(J)-B1(I,J)*Y(MPM+J)
    RES(I)=SUM
    TEM=DABS(RES(I))
    IF(TEM .GT. RABS) RABS=TEM
700  CONTINUE
800  CALL FF (X,Y,N,F)
    DO 900 I=2,N
    K1=(I-1)*M
    DO 900 J=1,M
    K1J=K1+J
    K1JM=K1J-M
    RES(K1J)=-Y(K1J)+Y(K1JM)+H/2*(F(K1J)+F(K1JM))+SK(K1JM)
    TEM=DABS(RES(K1J))

```

```

SYSS2400
SYSS2410
SYSS2420
SYSS2430
SYSS2440
SYSS2450
SYSS2460
SYSS2470
SYSS2480
SYSS2490
SYSS2500
SYSS2510
SYSS2520
SYSS2530
SYSS2540
SYSS2550
SYSS2560
SYSS2570
SYSS2580
SYSS2590
SYSS2600
SYSS2610
SYSS2620
SYSS2630
SYSS2640
SYSS2650
SYSS2660
SYSS2670
SYSS2680
SYSS2690
SYSS2700
SYSS2710
SYSS2720
SYSS2730
SYSS2740
SYSS2750
SYSS2760
SYSS2770
SYSS2780
SYSS2790
SYSS2800
SYSS2810
SYSS2820
SYSS2830
SYSS2840
SYSS2850
SYSS2860
SYSS2870
SYSS2880
SYSS2890
SYSS2900
SYSS2910
SYSS2920
SYSS2930
SYSS2940
SYSS2950
SYSS2960
SYSS2970
SYSS2980
SYSS2990

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```

          IF(ITEM .GT. RABS) RABS=TEM
900  CONTINUE
C
C      THE FIRST TIME THROUGH WE DON'T CHECK ANYTHING
C
    IF(INWT .EQ. 0) GO TO 950
910  IF(RABS .LT. REOLD .OR. INWT .EQ. 1) GO TO 520
C
C      IF THE RESIDUAL INCREASES AFTER THE FIRST ITERATION
C      WE ASSUME DIVERGENCE AND GO TO HALVE THE STEP SIZE
C
    DIVNEW=.TRUE.
    NU=NU+1
    GO TO 2600
920  IF(RABS .LE. EPS .OR. INWT .GE. 5) GO TO 1500
C.....
C      NEWTON EXIT (CONVERGENCE OR TOO MANY ITERATIONS)
C
950  CALL SYSLIN(M,N,X,Y,H,JACOB,RES,A1,B1,UU)
    REOLD=RABS
C
C      APPROXIMATE SOLUTION IS CORRECTED
C
1100 DO 1300 I=1,MPN
1300 Y(I)=Y(I)+UU(I)
C
C      NEXT TWO INSTRUCTIONS ARE FOR CONTROL OF PARAMETER
C      IN CONTINUATION METHOD
C
EPSNU=DMIN1(EPSNU+DELEPS,1.D0)
IF(EPSNU .GE. 1.00 .OR. DELEPS .LE. 0.D0) INWT=INWT+1
GO TO 410
C
C      CORRECTION AND ERROR CONTROL STARTS
C
1500  NU=NU+1
    NU2=2*NU+1
    AA(NU2)=-DFLCAT(NU)/DFLCAT((2**((2*NU-1)*NU2))
    AA(NU2+1)=0.D0
    CALL U2DCGS(NU,2,2,N1,M,AA,F,RES,IEPROR)
    IF(IEPROR .EQ. 1) GO TO 2600
    DO 1700 I=1,MPNM
    AUXI=RES(I)*H
    RES(I)=SK(I)-AUXI
    SK(I)=AUXI
1700  CONTINUE
    CALL SYSLIN(M,N,X,Y,H,JACOB,RES,A1,B1,UU)
C
C      ESTIMATE FOR MAX. ABSOLUTE ERROR (BY COMPONENTS)
C
ERRNEW=0.D0
DO 1900 J=1,M
1900  ABT(J)=0.D0
    DO 2100 I=1,M
    DO 2100 J=1,M
    KI=(I-1)*M+J
    UI=DABS(UU(KI))
    IF(UI .GT. ABT(J)) ABT(J)=UI
2100  CONTINUE
    DO 2300 J=1,M

```

```

SYSS3000
SYSS3010
SYSS3020
SYSS3030
SYSS3040
SYSS3050
SYSS3060
SYSS3070
SYSS3080
SYSS3090
SYSS3100
SYSS3110
SYSS3120
SYSS3130
SYSS3140
SYSS3150
SYSS3160
SYSS3170
SYSS3180
SYSS3190
SYSS3200
SYSS3210
SYSS3220
SYSS3230
SYSS3240
SYSS3250
SYSS3260
SYSS3270
SYSS3280
SYSS3290
SYSS3300
SYSS3310
SYSS3320
SYSS3330
SYSS3340
SYSS3350
SYSS3360
SYSS3370
SYSS3380
SYSS3390
SYSS3400
SYSS3410
SYSS3420
SYSS3430
SYSS3440
SYSS3450
SYSS3460
SYSS3470
SYSS3480
SYSS3490
SYSS3500
SYSS3510
SYSS3520
SYSS3530
SYSS3540
SYSS3550
SYSS3560
SYSS3570
SYSS3580
SYSS3590

```

```

IF(ABT(IJ) .GT. ERRNEW) ERRNEW=ABT(IJ)
2300 CONTINUE
      K=NU-1
2500 IF(ERRNEW .LE. TOL) RETURN
C
C<<<...      PRECISION ACHIEVED      .....
C
C      CC(K+1) CONTAINS ESTIMATED ERROR FOR CORRECTION K
C      ON MESH SIZE H/2 (UNDER THE HYPOTHESIS : ERROR IN
C      CORRECTION K(H) = C*H**[2*K+2]).
C
C      CC(NU)=ERRNEW*.4.D0**[1-NU]
C      IF(ERRNEW .LE. .1*ERROLD) GO TO 2550
C      IF(ERRNEW .GT. C1*ERROLD) GO TO 2600
C      C1=.5D0*C1
C
C      EITHER KEEP CORRECTING ...
C
C*****
C*
C* THE ERROR REDUCTION THRESHOLD C1 IS SET ORIGINALLY (AND
C* ARBITRARILY) TO 0.8. IF C1*ERROLD < ERRNEW WE HALVE
C* THE STEP. EACH TIME THAT 0.1*ERROLD < ERRNEW < C1*ERROLD
C* WE SET C1 TO 0.5*C1, THUS ACTUALLY ALLOWING THIS TO HA-
C* PPEEN A MAXIMUM OF THREE TIMES, BEFORE THE MORE STRICT
C* TEST WITH 0.1*ERROLD TAKES OVER COMPLETELY.
C* ERROLD IS THE ERROR ESTIMATE FOR THE LAST CORRECTION BUT
C* ONE, WHILE ERRNEW IS THE ONE CORRESPONDING TO THE LAST
C* CORRECTION.
C*
C*****
2550 ERROLD=ERRNEW
      EPS=DMAX1(EPSMAC,.01D0*H**2*ERROLD)
      GO TO 405
C
C.....>>>      OR HALVE THE STEP SIZE      .....
C
2600 IF(2*N-1 .LE. NMAX) GO TO 2625
      JERROR=2
      RETURN
C
C<<<...      TOO MANY GRID POINTS      .....
C
2625 N=2*N-1
      MPN=M*N
C
C      IF NEWTON DIVERGED WE START AGAIN WITH NU=0
C      IF(DIVNEW) GO TO 40
C*****
C* NOW WE DECIDE THE LEVEL OF CORRECTION ON THE NEW GRID
C* WE ASSUME THAT THE LAST ESTIMATED ERROR (PRESENTLY IN
C* ERROLD) WILL BE PRESERVED AFTER INTERPOLATING, AND
C* THEREFORE WE LOCATE THE FIRST INDEX I FOR WHICH
C* CC(I)*ERROLD, WHERE CC(I) IS THE PREDICTED ERROR FOR
C* THE (I-1) CORRECTION ON THE NEW GRID.
C*
C*****
      NUINT=NU

```

```

SYSS3600
SYSS3610
SYSS3620
SYSS3630
SYSS3640
SYSS3650
SYSS3660
SYSS3670
SYSS3680
SYSS3690
SYSS3700
SYSS3710
SYSS3720
SYSS3730
SYSS3740
SYSS3750
SYSS3760
SYSS3770
SYSS3780
SYSS3790
SYSS3800
SYSS3810
SYSS3820
SYSS3830
SYSS3840
SYSS3850
SYSS3860
SYSS3870
SYSS3880
SYSS3890
SYSS3900
SYSS3910
SYSS3920
SYSS3930
SYSS3940
SYSS3950
SYSS3960
SYSS3970
SYSS3980
SYSS3990
SYSS4000
SYSS4010
SYSS4020
SYSS4030
SYSS4040
SYSS4050
SYSS4060
SYSS4070
SYSS4080
SYSS4090
SYSS4100
SYSS4110
SYSS4120
SYSS4130
SYSS4140
SYSS4150
SYSS4160
SYSS4170
SYSS4180
SYSS4190

```

```

IF(ERRNEW .GE. ERROLD) GO TO 2650
ERROLD=ERRNEW
2650 DO 2700 I=1,NU
      IF(ERROLD .LT. CC(I)) GO TO 2700
      GO TO 2750
2700 CONTINUE
2750 NU=I-1
      EPS=DMAX1(EPSMAC,.005D0*H**2*ERROLD)
C*****
C*
C* COMPUTATION OF FIRST APPROXIMATION FOR Y ON NEW GRID
C* BY MEANS OF U2DCGS WILL GIVE ORDER OF INTERPOLATION
C* (2*NU+2), WHERE NU IS THE LAST SUCCESSFUL CORRECTION
C* PERFORMED ON THE COARSER GRID.
C*
C*****
2800 ND2=(N+1)/2
      MPD2=M*ND2
      DO 2900 I=1,MPD2
2900 RES(I)=Y(I)
      ND21=ND2-1
      CALL U2DCGS(NUINT,2,0,ND21,M,BB,Y,SK,IERROR)
      DO 3100 I=1,ND21
      KI=(I-1)*M
      KI2=2*KI
      DO 3100 L=1,M
      Y(KI2+L)=RES(KI+L)
3100 Y(KI2+M+L)=SK(KI+L)
      DO 3200 L=1,M
3200 Y(MPN-M+L)=RES(MPD2-M+L)
      GO TO 120
C.....      START ON NEW GRID      .....
C
C      END
C
C*****
C*
C* SUBROUTINE SYSLIN(M,N,X,Y,H,JACOB,RES,A1,B1,UU)
C* IMPLICIT REAL*8(A-H,D-Z)
C* DIMENSION X(1),Y(1),UU(1),RES(1)
C*****
C*
C* *** SOLUTION OF LINEAR SYSTEM ***
C*
C*****
C*
C* FOLLOWING ARRAYS ARE WORKING AREAS. DIMENSIONS INVOLVED
C* ARE : MMAX=10 , MMAX+1 = 11 , MMAX+2=100 ,
C* MMAX*MMAX=2600, MMAX*(MMAX+1)=110
C*
C* DIMENSION T(110),U(10),S(10,10),VM(10,11),R(100),
C* V(2600,11),AUX(10,11),A(10,10),B(10,10)
C* DOUBLE PRECISION JAB(10,10)
C*
C*****
C*
C* IT SOLVES THE 2*2 BLOCK SYSTEM
C*
C* I A B I I X I I B O I
C* 1---1---1 1---1 = 1---1
C*

```

```

SYSS4200
SYSS4210
SYSS4220
SYSS4230
SYSS4240
SYSS4250
SYSS4260
SYSS4270
SYSS4280
SYSS4290
SYSS4300
SYSS4310
SYSS4320
SYSS4330
SYSS4340
SYSS4350
SYSS4360
SYSS4370
SYSS4380
SYSS4390
SYSS4400
SYSS4410
SYSS4420
SYSS4430
SYSS4440
SYSS4450
SYSS4460
SYSS4470
SYSS4480
SYSS4490
SYSS4500
SYSS4510
SYSS4520
SYSL 10
SYSL 20
SYSL 30
SYSL 40
SYSL 50
SYSL 60
SYSL 70
SYSL 80
SYSL 90
SYSL 100
SYSL 110
SYSL 120
SYSL 130
SYSL 140
SYSL 150
SYSL 160
SYSL 170
SYSL 180
SYSL 190
SYSL 200
SYSL 210
SYSL 220
SYSL 230
SYSL 240
SYSL 250
SYSL 260
SYSL 270

```

```

C*          I C I D I I X I I B I I          *
C* WHERE A IS M*M AND D IS (M*N)*(M*N) AND ALL THE OTHER *
C* BLOCKS HAVE THE APPROPRIATE DIMENSIONS. D IS BLOCK LOWER *
C* TRIANGULAR, WITH MAIN DIAGONAL BLOCKS R(I) AND *
C* SUB-DIAGONAL S(I), ALL OF SIZES M*M. *
C* C HAS ONLY THE FIRST BLOCK DIFFERENT FROM ZERO, AND B *
C* ONLY THE LAST BLOCK DIFFERENT FROM ZERO. *
C* *
C*****
CONTINUE
C*****
C* *
C*          *** OUTLINE OF THE METHOD *** *
C* *
C* FIRST WE FORM C*=(CIB1) AND THEN WE SOLVE THE MATRIX *
C* SYSTEM DV=C* (V*=(VM)) *
C* BY THE RECURSION: V*(0)=0, *
C* R(I) V*(J)=(C*(I,J)-S(I) V*(J-1)), J=1,...,N *
C* THESE LINEAR SYSTEMS ARE SOLVED BY A STANDARD GAUSSIAN *
C* ELIMINATION CODE (SUBROUTINE DGELG). *
C* FINALLY X0 IS THE SOLUTION OF THE LINEAR SYSTEM *
C* (A - B V) X0 = B0 - B W AND *
C* X = M - V X0 *
C* *
C*****
CONTINUE
C*****
C* *
C*          *** CAUTION *** *
C* *
C* THIS SUBROUTINE MANIPULATES SOME MATRICES AS ONE *
C* DIMENSIONAL ARRAYS. *
C* SUBROUTINES DGELG AND DARRAY ARE FROM THE IBM/360 *
C* SCIENTIFIC SUBROUTINES PACKAGE. *
C* THE SUBROUTINE DARRAY TRANSFORMS BETWEEN TYPES OF *
C* STORAGE. *
C* *
C*****
950 CALL JACOB(X(1),Y,JAB)
      M2=M*M
      MPN=M*N
      M1=M+1
C
C          *** SOLUTION OF D.V* = C* ***
C
      DO 1000 I=1,M
1000 T(M2+I)=RES(M+I)
      DO 1800 L=2,M
          K1=(L-1)*M
          DO 1100 J=1,M
1100 U(J)=Y(K1+J)
C
C          *** GENERATION OF JACOBIAN ***
C
      H2=.5D0*M
      DO 50 J=1,M
          K1=(J-1)*M
          DO 50 I=1,M
              S(I,J)=H2*JAB(I,J)

```

```

SYSL 280
SYSL 290
SYSL 300
SYSL 310
SYSL 320
SYSL 330
SYSL 340
SYSL 350
SYSL 360
SYSL 370
SYSL 380
SYSL 390
SYSL 400
SYSL 410
SYSL 420
SYSL 430
SYSL 440
SYSL 450
SYSL 460
SYSL 470
SYSL 480
SYSL 490
SYSL 500
SYSL 510
SYSL 520
SYSL 530
SYSL 540
SYSL 550
SYSL 560
SYSL 570
SYSL 580
SYSL 590
SYSL 600
SYSL 610
SYSL 620
SYSL 630
SYSL 640
SYSL 650
SYSL 660
SYSL 670
SYSL 680
SYSL 690
SYSL 700
SYSL 710
SYSL 720
SYSL 730
SYSL 740
SYSL 750
SYSL 760
SYSL 770
SYSL 780
SYSL 790
SYSL 800
SYSL 810
SYSL 820
SYSL 830
SYSL 840
SYSL 850
SYSL 860
SYSL 870

```

```

      IF(I.EQ. J) S(T,J)=S(I,J)+1.D0
50 CONTINUE
      CALL JACOB(X(1),U,JAB)
      DO 60 I=1,M
          DO 60 J=1,M
              KI=(J-1)*M+I
              R(KI)=-H2*JAB(I,J)
              IF(I.EQ. J) R(KI)=R(KI)+1.D0
60 CONTINUE
      IF(L.NE. 2) GO TO 1300
      DO 1200 J=1,M
          K1=(J-1)*M
          DO 1200 I=1,M
1200 T(K1+I)=-S(I,J)
      GO TO 1700
C
C          COMPUTATION OF (C* - S.V*)
C
1300 DO 1500 K1=1,M1
      DO 1500 I=1,M
          SUM=0.D0
          DO 1400 J=1,M
1400 SUM=SUM+S(I,J)*VM(J,K1)
1500 T((K1-1)*M+I)=SUM
          DO 1600 I=1,M
1600 T(M2+I)=T(M2+I)+RES((L-1)*M+I)
1700 CALL DGELG (T,R,M,M1,1.D-7,IER)
1750 CALL DARRAY (I,M,M1,10,11,T,VM)
          DO 1800 J=1,M1
              DO 1800 I=1,M
1800 V((L-1)*M+I,J)=VM(I,J)
C
C          *** END OF RECURSION ***
C
      DO 2000 J=1,M1
          DO 2000 I=1,M
              SUM=0.D0
C
C          PRODUCTS B.V AND B.W
C
          DO 1900 K=1,M
1900 SUM=SUM+B(I,K)*VM(K,J)
2000 AUX(I,J)=SUM
C
C          (A - B.V)
C
      DO 2100 J=1,M
          K1=(J-1)*M
          DO 2100 I=1,M
2100 R(K1+I)=A1(I,J)-AUX(I,J)
          DO 2200 I=1,M
2200 U(I)=RES(I)-AUX(I,M1)
C
C          SOLUTION OF LINEAR SYSTEM
C          (A - B.V) X0 = (B0 - B.W)
C          AND COMPUTATION OF X
C
2210 CALL DGELG(U,R,M,1,1.D-7,IER)
2250 DO 2400 I=M1,MPN
          SUM=V(I,M1)

```

```

SYSL 980
SYSL 990
SYSL 900
SYSL 910
SYSL 920
SYSL 930
SYSL 940
SYSL 950
SYSL 960
SYSL 970
SYSL 980
SYSL 990
SYSL1000
SYSL1010
SYSL1020
SYSL1030
SYSL1040
SYSL1050
SYSL1060
SYSL1070
SYSL1080
SYSL1090
SYSL1100
SYSL1110
SYSL1120
SYSL1130
SYSL1140
SYSL1150
SYSL1160
SYSL1170
SYSL1180
SYSL1190
SYSL1200
SYSL1210
SYSL1220
SYSL1230
SYSL1240
SYSL1250
SYSL1260
SYSL1270
SYSL1280
SYSL1290
SYSL1300
SYSL1310
SYSL1320
SYSL1330
SYSL1340
SYSL1350
SYSL1360
SYSL1370
SYSL1380
SYSL1390
SYSL1400
SYSL1410
SYSL1420
SYSL1430
SYSL1440
SYSL1450
SYSL1460
SYSL1470

```

```

DO 2300 J=1,M          SYSL148C
2300 SUM=SUM-V(I,J)*UU(I,J)  SYSL149C
2400 UU(I)=SUM          SYSL1500
RETURN                SYSL1510
END                  SYSL1520
C
C .....U2DC 20
C .....U2DC 30
SUBROUTINE U2DCGS(K,P,Q,N,M,A,Y,S,IERROR)  U2DC 40
IMPLICIT REAL*8(A-H,O-Z)                U2DC 5C
INTEGER P,Q                               U2DC 60
DIMENSION A(50),Y(2600),S(2600),C(50)   U2DC 70
*****U2DC 80
C *
C * THIS IS A TWO POINT BOUNDARY VALUE DEFERRED CORRECTION GENERA-U2DC 100
C * TOR FOR SYSTEMS OF M EQUATIONS. GIVEN THE ASYMPTOTIC EXPANSION-U2DC 110
C *  $T(K) = \text{SUM}(A(IJ)+D*(J-1))Y(IJ-I)$  *U2DC 120
C *  $J = Q+1, \dots, Q+P+K$  *U2DC 130
C * AND VECTOR FUNCTION VALUES  $Y(I), \dots, Y(N+1)$ , CORRESPONDING TO *U2DC 140
C * AN UNIFORMLY H-SPACED MESH:  $X(I) = X(1) + (I-1)*H$ ,  $I=1, \dots, N+1$  *U2DC 150
C * U2DCGS WILL PRODUCE  $S(1), \dots, S(N-1)$ ; AN  $N*(Q+P+K)$  ORDER *U2DC 16C
C * APPROXIMATION TO  $T(K)$  AT MIDWAY BETWEEN EACH PAIR OF CONSEC-U2DC 170
C * UTIVE GRID POINTS *U2DC 180
C * FOR FIXED INTEGERS N,P,Q A RESTRICTION ON K IS *U2DC 190
C ***** K .LE. (N+1-Q)/P *****U2DC 200
C * ALSO P .GE. 1 , K .GE. 1 *U2DC 210
C * IERROR = 1 MEANS THAT ONE OF THESE CONDITIONS HAVE BEEN VIOL-U2DC 220
C * ATED AND NO CORRECTION HAS BEEN COMPUTED. A(1),...,A(Q) ARE SET *U2DC 230
C * TO ZERO BY U2DCGS. *U2DC 240
C * BOTH Y AND S ARE STORED AS VECTORS:  $Y(1), X(1), Y(2), X(1), \dots$  *U2DC 250
C * *U2DC 260
C * FOR MORE DETAILS SEE CHAPTER III OF "HIGH ORDER FINITE DIFFE-U2DC 270
C * RENCE SOLUTION OF DIFFERENTIAL EQUATIONS" BY V. PEREYRA. TECHN-U2DC 280
C * REP. STAN CS-73-348 , STANFORD UNIVERSITY (1973). *U2DC 290
C * *U2DC 300
C * APRIL 1973 ***** M. LENTINI P V. PEREYRA *****U2DC 310
C * *U2DC 320
C *****U2DC 330
IFIK .GT. (N+1-Q)/P .OR. P .LT. 1 .OR. K .LT. 1) GO TO 100  U2DC 340
IF(Q .EQ. 0) GO TO 10  U2DC 350
DO 20 I=1,Q  U2DC 360
20 A(I)=0.  U2DC 370
10 KK1=Q+P+K  U2DC 380
KK=KK1-1  U2DC 390
KMID=KK1/2  U2DC 400
IERROR=0  U2DC 410
KMID1=KMID-1  U2DC 420
C
C UNSYMMETRIC APPROXIMATION LEFT BOUNDARY  U2DC 440
C
1 IF(KMID1 .LT. 1) GO TO 25  U2DC 460
DO 5 I=1,KMID1  U2DC 470
CALL COEGEN(KK1,I,C,A)  U2DC 480
DO 7 L=1,M  U2DC 490
ACUM=0.  U2DC 500
DO 4 J=1,KK1  U2DC 510
4 ACUM=ACUM+C(I,J)*Y((J-1)*M+L)  U2DC 520
IT=(I-1)*M+L  U2DC 530
7 S(IT)=ACUM  U2DC 540
5 CONTINUE  U2DC 550

```

```

C
C CENTER RANGE  U2DC 568
C
C
25 CALL COEGEN(KK1,KMID,C,A)  U2DC 578
NF=N+1-KK1+KMID  U2DC 580
DO 40 I=KMID,NF  U2DC 590
II=I-KMID  U2DC 600
DO 39 L=1,M  U2DC 610
ACUM=0.  U2DC 620
DO 38 J=1,KK1  U2DC 630
38 ACUM=ACUM+C(I,J)*Y((II+J-1)*M+L)  U2DC 640
IT=(I-1)*M+L  U2DC 650
39 S(IT)=ACUM  U2DC 660
40 CONTINUE  U2DC 670
C
C RIGHT BOUNDARY  U2DC 680
C
C
KMIDP1=KMID+1  U2DC 700
DO 50 I=KMIDP1,KK  U2DC 710
CALL COEGEN(KK1,I,C,A)  U2DC 720
II=N-KK  U2DC 730
DO 49 L=1,M  U2DC 740
ACUM=0.  U2DC 750
DO 48 J=1,KK1  U2DC 760
48 ACUM=ACUM+C(I,J)*Y(II+J-1)*M+L)  U2DC 770
IT=(I+II-1)*M+L  U2DC 780
49 S(IT)=ACUM  U2DC 790
50 CONTINUE  U2DC 800
RETURN  U2DC 810
100 IERROR=1  U2DC 820
RETURN  U2DC 830
END  U2DC 840
C
C *****COEG 10
C *****COEG 20
C *****COEG 30
SUBROUTINE COEGEN(M,NP,C,BB)  COEG 40
IMPLICIT REAL*8(A-H,O-Z)  COEG 50
DIMENSION C(50),BB(50),ALF(50)  COEG 60
*****COEG 70
C * THIS IS A SLIGHTLY MODIFIED VERSION IN FORTRAN IV OF THE ALGOL *COEG 80
C * PROCEDURE PVAND , P. 901 OF "SOLUTION OF VANDERMONDE SYSTEMS *COEG 90
C * OF EQUATIONS" BY A. BJORCK AND V. PEREYRA MATH. COMP. VOL. 24 *COEG 100
C * PP. 893-903 (1970); WHERE A COMPLETE DESCRIPTION OF THE METHOD *COEG 110
C * USED CAN BE FOUND. *COEG 120
C *****COEG 130
DO 1 I=1,N  COEG 140
1 C(I)=BB(I)  COEG 150
DO 11 I=1,N  COEG 160
11 ALF(I)=I-NP-0.5D0  COEG 170
2 NN=N-1  COEG 180
DO 6 I=1,NN  COEG 190
LL=N-I  COEG 200
DO 6 J=1,LL  COEG 210
K=N-J+1  COEG 220
6 C(K)=C(K)-ALF(I)*C(K-1)  COEG 230
DO 8 I=1,NN  COEG 240
K=N-I  COEG 250
XKIN=1.D0/K  COEG 260
KM1=K-1  COEG 270
DO 8 J=KM1,N  COEG 280

```





```

      TB=PIVI*A(ILL)
      A(ILL)=A(L)
13  A(L)=TB
C
C   SAVE COLUMN INTERCHANGE INFORMATION
      A(LST)=J
C
C   ELEMENT PRODUCTION AND NEXT PIVOT SEARCH
      PIV=0.00
      LST=LST+1
      J=0
      DO 16 II=LST,LEND
        PIVI=-A(II)
        IST=II+M
        J=J+1
        DO 15 L=IST,MM,M
          LL=L-J
          A(L)=A(L)+PIVI*A(ILL)
          TB=DABS(A(L))
          IF(TB-PIV) 15,15,14
14  PIV=TB
      I=L
15  CONTINUE
      LL=L+J
16  R(LL)=R(LL)+PIVI*R(L)
17  LST=LST+M
C   END OF ELIMINATION LOOP
C
C   BACK SUBSTITUTION AND BACK INTERCHANGE
18  IF(M-1) 23,22,19
19  IST=MM+M
      LST=M+1
      DO 21 I=2,M
        II=LST-I
        IST=IST-LST
        L=IST-M
        L=A(L)+.500
        DO 21 J=II,MM,M
          TB=R(I)
          LL=J
          DO 20 K=IST,MM,M
            LL=LL+1
20  TB=TB-A(K)*R(LL)
          K=J+L
          R(J)=R(K)
21  R(K)=TB
22  RETURN
C
C   ERROR RETURN
23  IER=-1
      RETURN
      END
C
C   .....
C   SUBROUTINE DARRAY
C
C   PURPOSE
      CONVERT DATA ARRAY FROM SINGLE TO DOUBLE DIMENSION OR VICE
      VERSA. THIS SUBROUTINE IS USED TO LINK THE USER PROGRAM

```

```

DELG1160
DELG1170
DELG1180
DELG1190
DELG1200
DELG1210
DELG1220
DELG1230
DELG1240
DELG1250
DELG1260
DELG1270
DELG1280
DELG1290
DELG1300
DELG1310
DELG1320
DELG1330
DELG1340
DELG1350
DELG1360
DELG1370
DELG1380
DELG1400
DELG1410
DELG1420
DELG1430
DELG1440
DELG1450
DELG1460
DELG1470
DELG1480
DELG1490
DELG1500
DELG1510
DELG1520
DELG1530
DELG1540
DELG1550
DELG1560
DELG1570
DELG1580
DELG1590
DELG1600
DELG1610
DELG1620
DELG1630
DELG1640
DELG1650
DELG1660
DELG1670
DELG1680
DARR 10
DARR 20
DARR 30
DARR 40
DARR 50
DARR 60
DARR 70
DARR 80

```

```

C   WHICH HAS DOUBLE DIMENSION ARRAYS AND THE SSP SUBROUTINES
C   WHICH OPERATE ON ARRAYS OF DATA IN A VECTOR FASHION.
DARR 90
DARR 100
DARR 110
DARR 120
DARR 130
DARR 140
DARR 150
DARR 160
DARR 170
DARR 180
DARR 190
DARR 200
DARR 210
DARR 220
DARR 230
DARR 240
DARR 250
DARR 260
DARR 270
DARR 280
DARR 290
DARR 300
DARR 310
DARR 320
DARR 330
DARR 340
DARR 350
DARR 360
DARR 370
DARR 380
DARR 390
DARR 400
DARR 410
DARR 420
DARR 430
DARR 440
DARR 450
DARR 470
DARR 480
DARR 490
DARR 500
DARR 510
DARR 520
DARR 530
DARR 540
DARR 550
DARR 560
DARR 570
DARR 580
DARR 590
DARR 600
DARR 610
DARR 620
DARR 630
DARR 640
DARR 650
DARR 660
DARR 670
DARR 680
DARR 690

```

USAGE  
 CALL DARRAY (MODE,I,J,N,M,S,D)

DESCRIPTION OF PARAMETERS  
 MODE - CODE INDICATING TYPE OF CONVERSION  
       1 - FROM SINGLE TO DOUBLE DIMENSION  
       2 - FROM DOUBLE TO SINGLE DIMENSION  
 I - NUMBER OF ROWS IN ACTUAL DATA MATRIX  
 J - NUMBER OF COLUMNS IN ACTUAL DATA MATRIX  
 N - NUMBER OF ROWS SPECIFIED FOR THE MATRIX D IN  
       DIMENSION STATEMENT  
 M - NUMBER OF COLUMNS SPECIFIED FOR THE MATRIX D IN  
       DIMENSION STATEMENT  
 S - IF MODE=1, THIS VECTOR IS INPUT WHICH CONTAINS THE  
       ELEMENTS OF A DATA MATRIX OF SIZE I BY J. COLUMN I+1  
       OF DATA MATRIX FOLLOWS COLUMN I., ETC. IF MODE=2,  
       THIS VECTOR IS OUTPUT REPRESENTING A DATA MATRIX OF  
       SIZE I BY J CONTAINING ITS COLUMNS CONSECUTIVELY.  
       THE LENGTH OF S IS IJ, WHERE IJ=I\*J.  
 D - IF MODE=1, THIS MATRIX OF SIZE N BY M IS OUTPUT,  
       CONTAINING A DATA MATRIX OF SIZE I BY J IN THE FIRST  
       I ROWS AND J COLUMNS. IF MODE=2, THIS N BY M MATRIX  
       IS INPUT CONTAINING A DATA MATRIX OF SIZE I BY J IN  
       THE FIRST I ROWS AND J COLUMNS.

REMARKS  
 VECTOR S CAN BE IN THE SAME LOCATION AS MATRIX D. VECTOR S  
 IS REFERRED AS A MATRIX IN OTHER SSP ROUTINES, SINCE IT  
 CONTAINS A DATA MATRIX.  
 THIS SUBROUTINE CONVERTS ONLY GENERAL DATA MATRICES (STORAGE  
 MODE OF 0).

SUBROUTINES AND FUNCTION SUBROUTINES REQUIRED  
 NONE

METHOD  
 REFER TO THE DISCUSSION ON VARIABLE DATA SIZE IN THE SECTION  
 DESCRIBING OVERALL RULES FOR USAGE IN THIS MANUAL.

.....

SUBROUTINE DARRAY (MODE,I,J,N,M,S,D)  
 IMPLICIT REAL\*8(A-H,O-Z)  
 DIMENSION S(1),D(1)  
 NI=N-I

TEST TYPE OF CONVERSION  
 IF(MODE-1) 100, 100, 120

CONVERT FROM SINGLE TO DOUBLE DIMENSION

```

100 IJ=I*J+1
    NM=N*J+1
    DO 110 K=1,J
      NM=NM-NI
      DO 110 L=1,I
        IJ=IJ-1

```

<pre> NM=NM-1 110 DIMM1=S(IJ) GO TO 140 C       CONVERT FROM DOUBLE TO SINGLE DIMENSION C 120 IJ=0     NP=0     DO 130 K=1,J       DO 125 L=1,I         IJ=IJ+1         NM=NM+1 125 S(IJ)=D(NM) 130 NM=NM+NI C 140 RETURN     END </pre>	<pre> DARR 700 DARR 710 DARR 720 DARR 730 DARR 740 DARR 750 DARR 760 DARR 770 DARR 780 DARR 790 DARR 800 DARR 810 DARR 820 DARR 830 DARR 840 DARR 850 DARR 860 </pre>
--	---

```

C ..... MAIN0010
C ..... MAIN0020
C ..... MAIN0030
C ..... MAIN0040
C ..... MAIN0050
C ..... MAIN0060
C ..... MAIN0070
C ..... MAIN0080
C ..... MAIN0090
C ..... MAIN0100
C ..... MAIN0110
C ..... MAIN0120
C ..... MAIN0130
C ..... MAIN0140
C ..... MAIN0150
C ..... MAIN0160
C ..... MAIN0170
C ..... MAIN0180
C ..... MAIN0190
C ..... MAIN0200
C ..... MAIN0210
C ..... MAIN0220
C ..... MAIN0230
C ..... MAIN0240
C ..... MAIN0250
C ..... MAIN0260
C ..... MAIN0270
C ..... MAIN0280
C ..... MAIN0290
C ..... MAIN0300
C ..... MAIN0310
C ..... MAIN0320
C ..... MAIN0330
C ..... MAIN0340
C ..... MAIN0350
C ..... MAIN0360
C ..... MAIN0370
C ..... MAIN0380
C ..... MAIN0390
C ..... MAIN0400
C ..... MAIN0410
C ..... MAIN0420
C ..... MAIN0430
C ..... MAIN0440
C ..... MAIN0450
C ..... MAIN0460
C ..... MAIN0470
C ..... MAIN0480
C ..... MAIN0490
C ..... MAIN0500
C ..... MAIN0510
C ..... MAIN0520
C ..... MAIN0530
C ..... MAIN0540
C ..... MAIN0550
C ..... MAIN0560
C ..... MAIN0570
C ..... MAIN0580
C ..... MAIN0590

SAMPLE MAIN PROGRAM FOR VARIABLE FLUID PROPERTY PROBLEM.
PURPOSE
1) SOLVE THE DIFFERENTIAL EQUATIONS FOR VARIABLE FLUID
PROPERTY PROBLEM IN LAMINAR FREE CONVECTION ON A
VERTICAL FLAT PLATE. 2) CALCULATE HEAT TRANSFER COEFFICIENT.
SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED
PROP
FF2
JACOB2
SYSSOL
BLOCK DATA
COEFF
HEAT

IMPLICIT REAL*8(A-H,O-Z)
EXTERNAL FF2,JACOB2
COMMON /CCOEFF/ CA,CB,CC,CD,CE,CF,CG,CH,CI,CJ,CK,CL,CM,CN,CP,CR,
CS,CT,CU,CM,CAA,CBB,CCC,CCD,CEE,CFF,CGG,CHH
COMMON /CPROP/DENS(50),CPH(50),VISC(50),COND(50),TEMP(50),DTEMP
COMMON /TERM/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
COMMON /CHEAT/ QWX,HMX,RNUGR,GRX,CNUX,RATIO,Y,PRM,PRIN
COMMON /CTEMP/ TIN,TM,DT
COMMON /EQU/ M
DIMENSION A1(10,10),B1(10,10),ALPHA(10),Y(2600),X(521),ABT(10)

READ PROBLEM PARAMETERS.

THE FOLLOWING SUBROUTINE REQUIRES DATA INPUT, FOR SPECIFICATION
OF INPUT DATA REFER TO THE LISTING OF SUBROUTINE PROP.

CALL PROP

TM IS THE WALL TEMPERATURE IN DEG. R.
TIN IS THE FREE STREAM TEMPERATURE IN DEG. R.
DT IS THE TEMPERATURE DIFFERENCE IN DEG. R.
READ(5,1) TM,TIN
DT=TM-TIN

THE FOLLOWING INFORMATION IS REQUIRED FOR SUBROUTINE SYSSOL,
FOR THE DEFINITION OF THE PARAMETERS REFER TO THE LISTING OF
SUBROUTINE SYSSOL.

ND=4
TOL=1.D-6
N=NO
DELEPS=0.D0
M=5
A=0.D0
B=10.0000
DO 2000 I=1,M
ALPHA(I)=0.D0

```

```

DO 2000 J=1,M
A1(I,J)=0.D0
2000 B1(I,J)=0.D0
A1(1,1)=1.D0
A1(2,2)=1.D0
A1(4,4)=1.D0
B1(3,2)=1.D0
B1(5,4)=1.D0
ALPHA(4)=1.D0

CALL COEFF
CALL SYSSOL(M,N,A,B,ALPHA,A1,B1,TOL,DELEPS,X,Y,ABT,FF2,JACOB2,
JERROR)

WRITE(6,200)
WRITE(6,16) M,N,A,B,(ALPHA(I),I=1,M)
WRITE(6,18)((A1(I,J),J=1,M),I=1,M)
WRITE(5,24) ((B1(I,J),J=1,M),I=1,M)
WRITE(6,19) TOL
WRITE(6,13) (ABT(J),J=1,M)
WRITE(6,23) JERROR
WRITE(6,200)
WRITE(6,21)
DO 15 J=1,N,16
KA=M*(J-1)+1
KB=M*J
WRITE(6,22) J,X(I),Y(K),(K=KA,KB)
15 CONTINUE
WRITE(6,200)
WRITE(6,41) TM,TIN,DT
CALL HEAT
WRITE(6,42) PRM,PRIN
WRITE(6,43) CNUX,GRX
WRITE(6,30)
WRITE(6,31)
WRITE(6,32)
WRITE(6,33)
WRITE(6,34) QWX,HMX,RNUGR

1 FORMAT(2D12.5)
13 FORMAT(1H0,' ERROR ESTIMATE FOR COMPONENTS '//5D12.3)
16 FORMAT(1H0,' NUMBER OF EQUATIONS ',I3,'/' FINAL NUMBER OF MESH POINTS',I10)
SNTS ',I5,'/' LEFT BOUNDARY POINT = ',F7.2,5X,'RIGHT BOUNDARY POINT',I10
$ = ',F7.2,'/' BOUNDARY CONDITIONS'//5(F7.2,2X)
18 FORMAT(1H0,' MATRIX OF LEFT BOUNDARY CONDITIONS '//5(F7.2,2X))
20 FORMAT(1H0,' MATRIX OF RIGHT BOUNDARY CONDITIONS '//5(F7.2,2X))
19 FORMAT(1H0,' TOLERANCE ',D12.2)
21 FORMAT(1H0,'//7X,'M',7X,'ETA',15X,'F',15X,'FP',15X,'T',15X,'PAIN1070
$','TP')
22 FORMAT(1H0,3X,I4,3X,6(D12.5,5X))
23 FORMAT(1H0,' JERROR = ',I1)
30 FORMAT(1H0,'//14X,'Q*X**'(1/4) = HEAT TRANSFER IN BTU/HR-FT**'(7/4)')
31 FORMAT(1H0,13X,'H*X**'(1/4) = COEFFICIENT OF HEAT TRANSFER IN BTU/HMAIN1120
SR-FT**'(7/4)-R')
32 FORMAT(1H0,13X,'NU/GR = RATIO OF NUSSELT NO. AND GRASHOF NO.')
33 FORMAT(1H0,'//16X,'Q*X**'(1/4)',10X,'H*X**'(1/4)',12X,'NU/GR')
34 FORMAT(1H0,13X,E15.8,5X,E15.8,5X,E15.8)
41 FORMAT(1H0,' WALL TEMPERATURE = ',F7.2,' R'/' FREE STREAM TEMPRMAIN1170
SATURE = ',F7.2,' R'/' TEMPERATURE DIFFERENCE = ',F6.2,' R'/' MAIN1180
42 FORMAT(1H0,' WALL PRANDTL NUMBER = ',F6.2,'/' FREE STREAM PRANDTL MAIN1190

```

```

NUMBER = ,F6.2//)
43 FORMAT(1H0,' NUSSELT NUMBER = ',D12.5//,' GRASHOF NUMBER = ',D12.5//)
150 FORMAT(15.5,F15.5,F15.5,D15.5,D15.5)
200 FORMAT(1H1,' VARIABLE PROPERTY PROBLEM--REFRIGERANT 114')
C
STDP
END
C
SUBROUTINE FF2(X,Y,N,FF)
PURPOSE
THIS SUBROUTINE EVALUATES THE FIRST ORDER DIFFERENTIAL
EQUATIONS ON THE WHOLE GRID. FOR FURTHER INFORMATION SEE
THE LISTING OF SUBROUTINE SYSSOL AND EQUATIONS (5.13) AND
(5.14) OF THE THESIS.
C
SUBROUTINE FF2(X,Y,N,FF)
IMPLICIT REAL*8(A-H,O-Z)
COMMON /CCOEFF/ CA,CB,CC,CD,CE,CF,CG,CH,CI,CJ,CK,CL,CM,CN,CP,CR,
* CS,CT,CU,CV,CAA,CBB,CCC,CDD,CEE,CFF,CGG,CHH
DIMENSION X(1),Y(1),FF(1)
C
DO 10 I=1,N
K=(I-1)*5+1
FF(K)=Y(K+1)
FF(K+1)=Y(K+2)
FF(K+2)=-(CA+CB*Y(K+3)+CC*Y(K+3)**2+CD*Y(K+3)**3)*Y(K+4)*Y(K+2)
$ -(CE+CF*Y(K+3)+CG*Y(K+3)**2+CH*Y(K+3)**3)*Y(K+1)*Y(K+2)
$ +(CI+CJ*Y(K+3)+CK*Y(K+3)**2+CL*Y(K+3)**3)*Y(K+1)**2
$ -(CM+CN*Y(K+3)+CP*Y(K+3)**2+CR*Y(K+3)**3)
$ +(CS+CT*Y(K+3)+CU*Y(K+3)**2+CV*Y(K+3)**3)
FF(K+3)=Y(K+4)
10 FF(K+4)=-(CAA+CBB*Y(K+3)+CCC*Y(K+3)**2+CDD*Y(K+3)**3)*Y(K+4)**2
$ -(CEE+CFF*Y(K+3)+CGG*Y(K+3)**2+CHH*Y(K+3)**3)*Y(K+1)*Y(K+4)
C
RETURN
END
C
SUBROUTINE JACOB2(X,YX,JAB)
PURPOSE
THIS SUBROUTINE EVALUATES THE JACOBIAN OF THE DIFFERENTIAL
EQUATIONS AT A GIVEN GRID POINT. FOR FURTHER INFORMATION
SEE THE LISTING OF SUBROUTINE SYSSOL AND APPENDIX C
OF THE THESIS.
C
SUBROUTINE JACOB2(X,YX,JAB)
IMPLICIT REAL*8(A-H,O-Z)
COMMON /CCOEFF/ CA,CB,CC,CD,CE,CF,CG,CH,CI,CJ,CK,CL,CM,CN,CP,CR,
* CS,CT,CU,CV,CAA,CBB,CCC,CDD,CEE,CFF,CGG,CHH
COMMON /EQU/ P

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DIMENSION YX(1)
DOUBLE PRECISION JAB(10,10)
C
DO 10 I=1,M
DO 10 J=1,M
10 JAB(I,J)=0.00
JAB(1,2)=1.00
JAB(2,3)=1.00
JAB(3,1)=-CE*YX(3)-CF*YX(3)*YX(4)-CG*YX(3)*YX(4)**2
$ -CH*YX(3)*YX(4)**3
JAB(3,2)=2.00*CI*YX(2)+2.00*CJ*YX(2)*YX(4)+2.00*CK*YX(2)*YX(4)**2
$ +2.00*CL*YX(2)*YX(4)**3
JAB(3,3)=-CA*YX(5)-CB*YX(4)*YX(5)-CC*YX(5)*YX(4)**2
$ -CD*YX(5)*YX(4)**3-CE*YX(1)-CF*YX(1)*YX(4)
$ -CG*YX(1)*YX(4)**2-CH*YX(1)*YX(4)**3
JAB(3,4)=-CB*YX(3)*YX(5)-2.00*CC*YX(3)*YX(4)*YX(5)
$ -3.00*CD*YX(3)*YX(5)*YX(4)**2-CF*YX(1)*YX(3)
$ -2.00*CG*YX(1)*YX(3)*YX(4)-3.00*CH*YX(1)*YX(3)*YX(4)**2
$ +CJ*YX(2)**2+2.00*CK*YX(4)*YX(2)**2
$ +3.00*CL*YX(2)**2*YX(4)**2-CN*2.00*CP*YX(4)
$ -3.00*CR*YX(4)**2+CI+2.00*CU*YX(4)+3.00*CV*YX(4)**2
JAB(3,5)=-CA*YX(3)-CB*YX(3)*YX(4)-CC*YX(3)*YX(4)**2
$ -CD*YX(3)*YX(4)**3
JAB(4,5)=1.00
JAB(5,1)=-CEE*YX(5)-CFF*YX(5)*YX(4)-CGG*YX(5)*YX(4)**2
$ -CHH*YX(5)*YX(4)**3
JAB(5,4)=-CBB*YX(5)**2-2.00*CCC*YX(4)*YX(5)**2
$ -3.00*CDD*YX(4)**2*YX(5)**2-CFF*YX(1)*YX(5)
$ -2.00*CGG*YX(1)*YX(4)*YX(5)-3.00*CHH*YX(1)*YX(4)**2*YX(5)
JAB(5,5)=-2.00*CAA*YX(5)-2.00*CBB*YX(4)*YX(5)
$ -2.00*CCC*YX(4)**2*YX(5)-2.00*CDD*YX(5)*YX(4)**3
$ -CEE*YX(1)-CFF*YX(1)*YX(4)-CGG*YX(1)*YX(4)**2
$ -CHH*YX(1)*YX(4)**3
C
RETURN
END
C
BLOCK DATA
PURPOSE
THE COEFFICIENTS FOR THE CURVE FITTED TERMS AA1 , AA2 ,
AA3 , BB1 , BB2 ARE STORED IN THIS BLOCK DATA.FOR MORE
INFORMATION SEE EQUATIONS (5-1) THROUGH (5.5) OF THE THESIS.
THESE COEFFICIENTS SHOULD BE SUPPLIED BY THE USER.
C
BLOCK DATA
IMPLICIT REAL*8(A-H,O-Z)
COMMON /CONST/ A(25),B(25),C(25),D(25)
C
FLUID : REFRIGERANT-114
PRESSURE : 540 PSIA
TEMP. RANGE : FROM 774.69 TO 779.69 DEG. R
C
DATA A
1/- .10139867066526D05, -.282491577668854D05, -.65434837330367D03,
2- .7459939675048667D04, .2211013641874885D06/

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C      DATA B
      1/.3891121782649853D02,+.107838777841925003,+.248293511141094D01,
      2.2862293720560248D02,+.8461731580827346D03/
C      DATA C
      1/-.49775962134094D-01,+.1372623554228000,+.314340013846653D-02,
      2-.3660972533462388D-01,.1079552444084574D01/
C      DATA D
      1/.212258808660351D-04,+.5825886723323334D-04,+.13279202118396D-05,
      2.156092132755404D-04,+.4591322997601475D-03/
C      END
C      .....
C      SUBROUTINE COEFF
C      PURPOSE
C      THIS SUBROUTINE CALCULATES THE COEFFICIENTS OF THE REDUCED
C      DIFFERENTIAL EQUATIONS. COEFFICIENT FOR TERMS A1 , 3A2 ,
C      2A2 , A3 , B1 , 3B2 .SEE EQUATIONS (5.7) THROUGH (5.12)
C      OF THE THESIS FOR MORE INFORMATION.
C      .....
C      SUBROUTINE COEFF
      IMPLICIT REAL*8(A-H,O-Z)
      COMMON /CCOEFF/ CA,CB,CC,CD,CE,CF,CG,CH,CJ,CK,CL,CM,CN,CP,CR,
      * CS,CT,CU,CV,CA, CBB,CCC,CDD,CEE,CF,CGG,CHH
      COMMON/TERM/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
      COMMON/CPROP/DENSI(50),CPH(50),VISCI(50),COND(50),TEMP(50),DTEMP
      COMMON /CONST/ A1(25),B1(25),C1(25),D1(25)
      COMMON /CTEMP/ TIN,TM,DT
C      THE FOLLOWING IF STATEMENTS ARE FOR THE CURVE FITTED
C      TERMS AA1,AA2,AA3,BB1,BB2, AT DIFFERENT TEMP. INTERVALS
C      AND ONE SPECIFIC PRESSURE AND SHOULD BE SUPPLIED BY THE USER.
C      THE IF STATEMENTS SHOULD BE ARRANGED IN ORDER OF INCREASING TEMP.
C      FLUID : REFRIGERANT-114
C      PRESSURE : 540 PSIA
C      TEMP. RANGE : FROM 774.69 TO 779.69 DEG. R
C
C      IF(TIN .GE. 774.69 .AND. TIN .LE. 779.69) GO TO 10
C      IF(TIN .GT. 779.69 .AND. TIN .LE. 809.69) GO TO 20
C      " " " " " "
C      " " " " " "
C
C      IF THE NEXT STATEMENT IS READ THE TEMP. IS OUT OF RANGE.
C      GO TO 70
C
10 I=1
   GO TO 60
20 I=2
   GO TO 60
30 I=3
   GO TO 60
40 I=4
   GO TO 60
BLK00250
BLK00260
BLK00270
BLK00280
BLK00290
BLK00300
BLK00310
BLK00320
BLK00330
BLK00340
BLK00350
BLK00360
BLK00370
BLK00380
COEF0010
COEF0020
COEF0030
COEF0040
COEF0050
COEF0060
COEF0070
COEF0080
COEF0090
COEF0100
COEF0110
COEF0120
COEF0130
COEF0140
COEF0150
COEF0160
COEF0170
COEF0180
COEF0190
COEF0200
COEF0210
COEF0220
COEF0230
COEF0240
COEF0250
COEF0260
COEF0270
COEF0280
COEF0290
COEF0300
COEF0310
COEF0320
COEF0330
COEF0340
COEF0350
COEF0360
COEF0370
COEF0380
COEF0390
COEF0400
COEF0410
COEF0420
COEF0430
COEF0440
COEF0450
COEF0460
50 I=5
50 CONTINUE
C
J=(I-1)*5
A1=A(J+1)
A2=A(J+2)
A3=A(J+3)
A4=A(J+4)
A5=A(J+5)
C
B1=B(J+1)
B2=B(J+2)
B3=B(J+3)
B4=B(J+4)
B5=B(J+5)
C
C1=C(J+1)
C2=C(J+2)
C3=C(J+3)
C4=C(J+4)
C5=C(J+5)
C
D1=D(J+1)
D2=D(J+2)
D3=D(J+3)
D4=D(J+4)
D5=D(J+5)
C
TIN1=TIN
TIN2=TIN1+TIN
TIN3=TIN2+TIN
C
DT1=DT
DT2=DT1+DT
DT3=DT2+DT
DT4=DT3+DT
C
K=IDINT(((TM-TEMP(1))/DTEMP)+0.5)+1
JJ=IDINT(((TIN-TEMP(1))/DTEMP)+0.5)+1
C
CALCULATE COEFFICIENTS FOR TERM A1
CA=(A1+B1*TIN1+C1*TIN2+D1*TIN3)*DT1
CB=(B1+2.00*C1*TIN1+3.00*D1*TIN2)*DT2
CC=(C1+3.00*D1*TIN1)*DT3
CD=D1*DT4
C
CALCULATE COEFFICIENTS FOR TERM 3A2
DENVIS=DENSI(K)*VISCI(K)
CE=3.00*(A2+B2*TIN1+C2*TIN2+D2*TIN3)*DENVIS
CF=3.00*(B2+2.00*C2*TIN1+3.00*D2*TIN2)*DENVIS*DT1
CG=3.00*(C2+3.00*D2*TIN1)*DENVIS*DT2
CH=3.00*D2*DENVIS*DT3
C
CALCULATE COEFFICIENTS FOR TERM 2A2
CI=2.00*CE/3.00
CJ=2.00*CF/3.00
CK=2.00*CG/3.00
COEF0470
COEF0480
COEF0490
COEF0500
COEF0510
COEF0520
COEF0530
COEF0540
COEF0550
COEF0560
COEF0570
COEF0580
COEF0590
COEF0600
COEF0610
COEF0620
COEF0630
COEF0640
COEF0650
COEF0660
COEF0670
COEF0680
COEF0690
COEF0700
COEF0710
COEF0720
COEF0730
COEF0740
COEF0750
COEF0760
COEF0770
COEF0780
COEF0790
COEF0800
COEF0810
COEF0820
COEF0830
COEF0840
COEF0850
COEF0860
COEF0870
COEF0880
COEF0890
COEF0900
COEF0910
COEF0920
COEF0930
COEF0940
COEF0950
COEF0960
COEF0970
COEF0980
COEF0990
COEF1000
COEF1010
COEF1020
COEF1030
COEF1040
COEF1050
COEF1060

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C CL=2.00*CH/3.00
C CEF1070
C CEF1080
C CALCULATE COEFFICIENTS FOR TERM A3 CEF1090
C CEF1100
RATIO1=DENS(JJ)*VISC(K)*DENS(K)**2/(DENS(JJ)-DENS(K)) CEF1110
CM=(A3+B3)*TIN1+C3*TIN2+D3*TIN3)*RATIO1 CEF1120
CN=(B3+2.00*C3*TIN1+3.00*D3*TIN2)*RATIO1*DT1 CEF1130
CP=(C3+3.00*D3*TIN1)*RATIO1*DT2 CEF1140
CR=D3*RATIO1*DT3 CEF1150
C CEF1160
RATIO2=RATIO1/DENS(JJ) CEF1170
CS=(A2+B2*TIN1+C2*TIN2+D2*TIN3)*RATIO2 CEF1180
CT=(B2+2.00*C2*TIN1+3.00*D2*TIN2)*RATIO2*DT1 CEF1190
CU=(C2+3.00*D2*TIN1)*RATIO2*DT2 CEF1200
CW=D2*RATIO2*DT3 CEF1210
C CEF1220
C CALCULATE COEFFICIENTS FOR TERM B1 CEF1230
C CEF1240
CAA=(A4+B4*TIN1+C4*TIN2+D4*TIN3)*DT CEF1250
CBB=(B4+2.00*C4*TIN1+3.00*D4*TIN2)*DT2 CEF1260
CCC=(C4+3.00*D4*TIN1)*DT3 CEF1270
CDD=D4*DT4 CEF1280
C CEF1290
C CALCULATE COEFFICIENTS FOR TERM 3B2 CEF1300
C CEF1310
CEE=3.00*(A5+B5*TIN1+C5*TIN2+D5*TIN3)*DENVIS CEF1320
CFF=3.00*(B5+2.00*C5*TIN1+3.00*D5*TIN2)*DENVIS*DT1 CEF1330
CGG=3.00*(C5+3.00*D5*TIN1)*DENVIS*DT2 CEF1340
CHH=3.00*D5*DENVIS*DT3 CEF1350
C CEF1360
GD TO 80 CEF1370
70 WRITE (6,1) TIN CEF1380
1 FORMAT (1H0,5X,'TIN = ',F10.5,5X,'TEMPERATURE OUT OF RANGE') CEF1390
80 CONTINUE CEF1400
C CEF1410
RETURN CEF1420
END CEF1430
HEAT0010
C .....HEAT0020
C SUBROUTINE HEAT HEAT0030
C HEAT0040
C PURPOSE HEAT0050
C CALCULATE HEAT TRANSFER COEFFICIENT. HEAT0060
C REMARK HEAT0070
C FOR REFRIGERANT-114 STATEMENTS 10, 20, AND 30 SHOULD BE HEAT0080
C REPLACED BY THE FOLLOWING STATEMENTS. HEAT0090
C CONDM=CONDIJJ)*241.9000 HEAT0100
C PRIN=(VISC(IJJ)*CPH(IJJ)*(1.00-02))/CONDIJJ HEAT0110
C PRW=(VISC(IJ)*CPH(IJ)*(1.00-02))/CONDIJ HEAT0120
C .....HEAT0130
C HEAT0140
C HEAT0150
C HEAT0160
C HEAT0170
C HEAT0180
C HEAT0190
C HEAT0200
C HEAT0210
C HEAT0220
C HEAT0230
C HEAT0240
C HEAT0250
C HEAT0260
C HEAT0270
C HEAT0280
C HEAT0290
C HEAT0300
C HEAT0310
C HEAT0320
C HEAT0330
C HEAT0340
C HEAT0350
C HEAT0360
C HEAT0370
C HEAT0380
C HEAT0390
C HEAT0400
C HEAT0410
C HEAT0420
C HEAT0430
C HEAT0440
C HEAT0450
C HEAT0460
C HEAT0470
C HEAT0480
C HEAT0490
C HEAT0500
C HEAT0510
C HEAT0520
C HEAT0530
C HEAT0540
C HEAT0550
C HEAT0560
C HEAT0570
C HEAT0580
C HEAT0590
C HEAT0600
C HEAT0610
C HEAT0620
C HEAT0630
C HEAT0640
C HEAT0650
C HEAT0660

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C DIMENSION Y(2*00)
C HEAT0240
C HEAT0250
C CALCULATE CONSTANT C IN SIMILARITY VARIABLE ETA HEAT0260
C HEAT0270
C J=IDINT(((TM-TEMP(1))/DTEMP)+0.5)+1 HEAT0280
C JJ=IDINT(((TIN-TEMP(1))/DTEMP)+0.5)+1 HEAT0290
C DENR=(DENS(JJ)-DENS(IJ))/DENS(IJ) HEAT0300
C VTSKM=VISC(IJ)/(1490.000) HEAT0310
C VISKM=VISKM/DENS(IJ) HEAT0320
C VISKM2=VISKM*VISKM HEAT0330
C CI=(32.200*DENR)/(4.00*VISKM2) HEAT0340
C=C1**0.25 HEAT0350
C HEAT0360
C CALCULATE RATE OF HEAT TRANSFER HEAT0370
C HEAT0380
C TPW=Y(M) HEAT0390
10 CONDM=CONDIJ) HEAT0400
C QMX=-CONDM*(TM-TIN)*C*TPW HEAT0410
C HEAT0420
C CALCULATE HEAT TRANSFER COEFFICIENT HEAT0430
C HEAT0440
C HEAT0450
C HEAT0460
C HEAT0470
C HEAT0480
C HEAT0490
C HEAT0500
C HEAT0510
C HEAT0520
C HEAT0530
C HEAT0540
C HEAT0550
C HEAT0560
C HEAT0570
C HEAT0580
C HEAT0590
C HEAT0600
C HEAT0610
C HEAT0620
C HEAT0630
C HEAT0640
C HEAT0650
C HEAT0660

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VARIABLE PROPERTY PROBLEM--REFRIGERANT 114

VARIABLE PROPERTY PROBLEM--REFRIGERANT 114

NUMBER OF EQUATIONS 5		FINAL NUMBER OF MESH POINTS 97		LEFT BOUNDARY POINT = 0.0		RIGHT BOUNDARY POINT = 10.00	
BOUNDARY CONDITIONS		BOUNDARY CONDITIONS		BOUNDARY CONDITIONS		BOUNDARY CONDITIONS	
0.0	0.0	0.0	0.0	1.00	0.0	0.0	0.0
1.00	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.00	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	1.00	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
MATRIX OF RIGHT BOUNDARY CONDITIONS							
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	1.00	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	1.00	0.0	0.0	0.0
TOLERANCE 0.100-05							
ERRR ESTIMATE FOR COMPONENTS							
0.7540-06	0.1270-06	0.4332-07	0.3010-07	0.2750-07	0.2750-07	0.2750-07	0.2750-07
JERRR = 0							

N	ETA	F	FP	FPP	T	TP
1	0.0	0.0	0.132350-22	0.546940+00	0.100000+01	-0.854410+00
5	0.416470+00	0.358520-01	0.147180+30	0.288370+00	0.465020+30	-0.785340+90
9	0.833330+00	0.140820+00	0.180020+00	-0.594480-02	0.363680+00	-0.593300+00
13	0.125000+01	0.173510+00	0.159100+00	-0.782610-01	0.164230+00	-0.346830+00
17	0.166470+01	0.227340+00	0.122770+00	-0.884990-01	0.458470-01	-0.159780+00
21	0.208330+01	0.241170+00	0.885530-01	-0.739440-01	0.225470-01	-0.610110-01
25	0.250000+01	0.312220+00	0.416400-01	-0.551720-01	0.498280-02	-0.203120-01
29	0.291670+01	0.333600+00	0.421800-01	-0.390840-01	0.201370-02	-0.461440-02
33	0.333330+01	0.348160+00	0.285440-01	-0.270080-01	0.552630-03	-0.174030-02
37	0.375000+01	0.357980+00	0.191850-01	-0.184350-01	0.144420-03	-0.471500-02
41	0.416670+01	0.366560+00	0.128180-01	-0.125010-01	0.380230-04	-0.123840-03
45	0.458330+01	0.368950+00	0.850840-02	-0.844110-02	0.971050-05	-0.319490-04
49	0.500000+01	0.371450+00	0.560370-02	-0.568350-02	0.245400-05	-0.812430-05
53	0.541670+01	0.373750+00	0.365200-02	-0.360900-02	0.416490-06	-0.204810-05
57	0.583330+01	0.375680+00	0.234590-02	-0.254250-02	0.154150-06	-0.513400-06
61	0.625000+01	0.375700+00	0.147650-02	-0.168630-02	0.384400-07	-0.128220-06
65	0.666670+01	0.376250+00	0.902280-03	-0.110770-02	0.954900-08	-0.319490-07
69	0.708330+01	0.376540+00	0.527480-03	-0.716660-03	0.237950-08	-0.794960-08
73	0.750000+01	0.376710+00	0.287370-03	-0.452920-03	0.591310-09	-0.197640-08
77	0.791670+01	0.376800+00	0.136210-03	-0.274480-03	0.144820-09	-0.491130-09
81	0.833330+01	0.376840+00	0.504670-04	-0.154330-03	0.363490-10	-0.122020-09
85	0.875000+01	0.376890+00	0.415980-05	-0.730600-04	0.893050-11	-0.303130-10



VARIABLE PROPERTY PROBLEM--REFRIGERANT 114

WALL TEMPERATURE = 779.69 R

FREE STREAM TEMPERATURE = 774.69 R

TEMPERATURE DIFFERENCE = 5.00 R

WALL PRANDTL NUMBER = 2.55

FREE STREAM PRANDTL NUMBER = 3.55

NUSSELT NUMBER = 0.975780+03

GRASHOF NUMBER = 0.676230+13

Q\*\*{(1/4)} = HEAT TRANSFER IN BTU/HR-FT\*\*{(7/4)}

H\*\*{(1/4)} = COEFFICIENT OF HEAT TRANSFER IN BTU/HR-FT\*\*{(7/4)}-R

NU/GR = RATIO OF NUSSELT NO. AND GRASHOF NO.

Q**{(1/4)}	H**{(1/4)}	NU/GR
0.642449130+02	0.128489830+02	0.854408240+00

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C
C .....MAIN0020
C
C SAMPLE MAIN PROGRAM FOR CONSTANT PROPERTY PROBLEM.
C
C PURPOSE
C 1) SOLVE THE DIFFERENTIAL EQUATIONS FOR COSTANT FLUID
C PROPERTY PROBLEM IN LAMINAR FREE CONVECTION ON A
C VERTICAL FLAT PLATE, 2) CALCULATE HEAT TRANSFER CDEFFICIENT.
C
C SUBROUTINES AND FUNCTION SUBPROGRAMS REQUIRED
C PROP MAIN010
C FFI MAIN012
C JACOBI MAIN013
C SYSSOL MAIN014
C HEAT MAIN015
C .....MAIN018
C
C IMPLICIT REAL*8(A-H,O-Z)
C EXTERNAL FFI,JACOBI
C COMMON/CPROP/DENS(50),CPH(50),VISC(50),COND(50),TEMP(50),DTEMP
C COMMON/TERM/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
C COMMON /CHEAT/ QWX,HMX,RNUGR,GRX,CNUX,RATIO,Y,PRM,PRIN
C COMMON /CTEMP/ TIN,TM,DT,TX
C COMMON /EQU/ M
C COMMON /CONS/ PR
C DIMENSION A1(10,10),B1(10,10),ALPHA(10),Y(2600),X(521),ABT(10)
C
C READ PROBLEM PARAMETERS.
C
C THE FOLLOWING SUBROUTINE REQUIRES DATA INPUT, FOR SPECIFICATION
C OF INPUT DATA REFER TO THE LISTING OF SUBROUTINE PROP.
C
C CALL PROP
C
C TW IS THE WALL TEMPERATURE IN DEG. R.
C TIN IS THE FREE STREAM TEMPERATURE IN DEG. R.
C TX IS THE REFERENCE TEMPERATURE IN DEG. R.
C DT IS THE TEMPERATURE DIFFERENCE IN DEG. R.
C READ(5,1) TW,TIN,TX
C DT=TW-TIN
C
C CALCULATE PRANDTL NUMBER AT THE SPECIFIED REFERENCE TEMPERATURE.
C L=IDINT((TX-TEMP(1))/DTEMP)+0.5)+1
C PR=(VISC(L)*CPH(L)+1.0D-02)/COND(L)
C
C THE FOLLOWING INFORMATION IS REQUIRED FOR SUBROUTINE SYSSOL,
C FOR THE DEFINITION OF THE PARAMETERS REFER TO THE LISTING OF
C SUBROUTINE SYSSOL.
C
C N0=4
C TOL=1.0D-6
C M=N0
C DELEPS=0.00
C M=5
C A=0.00
C B=10.00

```

```

DO 1000 I=1,M
ALPHA(I)=0.00
DO 1000 J=1,M
A(I,J)=0.00
1000 B(I,J)=0.00
A(I,1)=1.00
A(I,2)=1.00
A(I,4)=1.00
B(I,3)=1.00
B(I,5)=1.00
ALPHA(4)=1.00
C
CALL SYSSOL(M,N,A,B,ALPHA,A1,B1,TOL,DELEPS,X,Y,ABT,FFI,JACOBI,
JERROR)
C
WRITE(6,100) PR
*PIE(6,16) M,A,A,B,(ALPHA(I),I=1,M)
WRITE(6,13)((A1(I,J),J=1,M),I=1,M)
WRITE(6,24)((B1(I,J),J=1,M),I=1,M)
WRITE(6,19) TOL
WRITE(6,13) (ABT(J),J=1,M)
WRITE(6,23) JERROR
WRITE(6,100) PR
DO 15 J=1,N,24
KA=M*(J-1)+1
KB=M*J
WRITE(6,22) J,X(J),(Y(K),K=KA,KB)
15 CONTINUE
WRITE(6,100) PR
WRITE(6,41) TW,TIN,DT,TX
CALL HEAT
WRITE(6,42) PRM,PRIN
WRITE(6,43) CNUX,GRX
WRITE(6,30)
WRITE(6,31)
WRITE(6,32)
WRITE(6,33)
WRITE(6,34) QWX,HMX,RNUGR
C
1 FORMAT(3D12.5)
13 FORMAT(1H0," ERROR ESTIMATE FOR COMPONENTS '//5D12.3)
16 FORMAT(1H0," NUMBER OF EQUATIONS ',I3//" FINAL NUMBER OF MESH POINTS',I3)
SNTS ',I5//" LEFT BOUNDARY POINT = ',F7.2,5X,"RIGHT BOUNDARY POINT=
$ = ',F7.2//" BOUNDARY CONDITIONS '//5(F7.2,2X))
18 FORMAT(1H0," MATRIX OF LEFT BOUNDARY CONDITIONS '//5(F7.2,2X))
24 FORMAT(1H0," MATRIX OF RIGHT BOUNDARY CONDITIONS '//5(F7.2,2X))
19 FORMAT(1H0," TOLERANCE ',D12.2)
21 FORMAT(1H0//7X,"A",7X,"ETA",15X,"F",15X,"FP",15X,"FPP",15X,"T",15X,"A1",1080
$,"T")
22 FORMAT(1H0,3X,I4,3X,6(D12.5,5X))
23 FORMAT(1H0," JERROR = ',I1)
70 FORMAT(1H0//14X,"G*X**(1/4) = HEAT TRANSFER IN BTU/HR-FT**(7/4)",7X,"A1",1120
31 FORMAT(1H0,13X,"H*X**(1/4) = COEFFICIENT OF HEAT TRANSFER IN BTU/HR",1130
SR-FT**(7/4)-R")
32 FORMAT(1H0,13X,"NU/GR = RATIO OF NUSSELT NO. AND GRASHOF NO.,")
33 FORMAT(1H0//16X,"G*X**(1/4)",10X,"H*X**(1/4)",12X,"NU/GR")
74 FORMAT(1H0,13X,E15.8,5X,E15.8,5X,E15.8)
41 FORMAT(1H0," WALL TEMPERATURE = ',F7.2," R'//" FREE STREAM TEMPPMAIN1180
$ATURE = ',F7.2," R'//" TEMPERATURE DIFFERENCE = ',F6.2," R'//" MAIN1190

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```

$ REFERENCE TEMPERATURE = ',F6.2,' 9'//)
42 FORMAT(1H0,' WALL PRANDTL NUMBER = ',F6.2'//) FREE STREAM PRANDTL
NUMBER = ',F6.2'//)
43 FORMAT(1H0,' NUSSLELY NUMBER = ',D12.5'//,' GRASHOF NUMBER = ',D12.5'//)
100 FORMAT(1H1,' REFRIGERANT 114 -- CONSTANT PROPERTY PROBLEM WITH D
S= ',F6.2)
C
STGP
END
C
.....
SUBROUTINE FF1(X,Y,N,FF)
C
PURPOSE
THIS SUBROUTINE EVALUATES THE FIRST ORDER DIFFERENTIAL
EQUATIONS ON THE WHOLE GRID. FOR FURTHER INFORMATION SEE
THE LISTING OF SUBROUTINE SYSSOL AND APPENDIX C OF
THE THESIS.
C
.....
SUBROUTINE FF1(X,Y,N,FF)
IMPLICIT REAL*8(A-H,O-Z)
COMMON /CONS/ PP
DIMENSION X(1),Y(1),FF(1)
C
DO 10 I=1,N
KI=(I-1)*5+1
FF(KI)=Y(KI+1)
FF(KI+1)=Y(KI+2)
FF(KI+2)=-3.00*Y(KI)*Y(KI+2)+2.00*Y(KI+1)*Y(KI+3)
FF(KI+3)=Y(KI+4)
10 FF(KI+4)=-3.00*PP*Y(KI)*Y(KI+4)
C
RETURN
END
C
.....
SUBROUTINE JACOB1(X,YX,JAB)
C
PURPOSE
THIS SUBROUTINE EVALUATES THE JACOBIAN OF THE DIFFERENTIAL
EQUATIONS AT A GIVEN GRID POINT. FOR FURTHER INFORMATION
SEE THE LISTING OF SUBROUTINE SYSSOL AND APPENDIX C
OF THE THESIS.
C
.....
SUBROUTINE JACGB1(X,YX,JAB)
IMPLICIT REAL*8(A-H,O-Z)
COMMON /CONS/ PP
COMMON /EQU/ M
DIMENSION YX(1)
DOUBLE PRECISION JAB(10,10)
C
DO 10 I=1,M
DO 10 J=1,M

```

```

MAIN1200
MAIN1210
MAIN1220
MAIN1230
MAIN1240
MAIN1250
MAIN1260
MAIN1270
MAIN1280
MAIN1290
FF100010
FF100020
FF100030
FF100040
FF100050
FF100060
FF100070
FF100080
FF100090
FF100100
FF100110
FF100120
FF100130
FF100140
FF100150
FF100160
FF100170
FF100180
FF100190
FF100200
FF100210
FF100220
FF100230
FF100240
FF100250
FF100260
FF100270
FF100280
JAC10010
JAC10020
JAC10030
JAC10040
JAC10050
JAC10060
JAC10070
JAC10080
JAC10090
JAC10100
JAC10110
JAC10120
JAC10130
JAC10140
JAC10150
JAC10160
JAC10170
JAC10180
JAC10190
JAC10200
JAC10210
JAC10220

```

```

10 JAB(I,J)=0.00
JAB(1,2)=1.00
JAB(2,3)=1.00
JAB(2,1)=-2.00*YX(1)
JAB(3,2)=4.00*YX(2)
JAB(3,3)=-3.00*YX(1)
JAB(3,4)=-1.00
JAB(4,5)=1.00
JAB(5,1)=-3.00*PP*YX(1)
JAB(5,5)=-3.00*PP*YX(1)
C
RETURN
END
C
.....
SUBROUTINE HEAT
C
PURPOSE
CALCULATE HEAT TRANSFER COEFFICIENT.
REMARK
FOR REFRIGERANT-114 STATEMENTS 10, 20, AND 30 SHOULD BE
REPLACED BY THE FOLLOWING STATEMENTS.
CONDX=CONDX(J)*241.0000
PRIN=(VISC(JJ)*CPH(JJ)*(1.00-02))/CONDX(JJ)
PRM=(VISC(JJ)*CPH(JJ)*(1.00-02))/CONDX(JJ)
C
.....
SUBROUTINE HEAT
IMPLICIT REAL*8(A-H,O-Z)
COMMON /CHEAT/ QWX,HWX,RNUGR,GRX,CNWX,RATIC,Y,PPR,PRIN
COMMON /CTEMP/ TIN,TM,DT,TA
COMMON /TERP/AA1(50),AA2(50),AA3(50),BB1(50),BB2(50)
COMMON /CPRDP/ DENSI(50),CPH(50),VISC(50),COND(50),TEMP(50),DTEMP
COMMON /EQU/ M
DIMENSION Y(2600)
C
CALCULATE CONSTANT C IN SIMILARITY VARIABLE ETA
C
J=IDINT(((TW-TEMP(1))/DTEMP)+0.5)+1
K=IDINT(((TX-TEMP(1))/DTEMP)+0.5)+1
JJ=IDINT(((TIN-TEMP(1))/DTEMP)+0.5)+1
DENP=(DENSI(JJ)-DENSI(1))/DENSI(J)
VISPX=VISC(K)/(1+90.00)
VISNX=VISC(K)/DENSI(K)
VISNX2=VISNX*VISNX
CIX=(32.200*DENP)/(4.00*VISNX2)
CX=CIX*0.25
C
CALCULATE RATE OF HEAT TRANSFER
C
TPWX=Y(M)
10 CONDX=CONDX(J)
QWX=-CONDX*(TW-TIN)*CX*TPWX
C
CALCULATE HEAT TRANSFER COEFFICIENT
C
HWX=QWX/(TW-TIN)

```

```

JAC10230
JAC10240
JAC10250
JAC10260
JAC10270
JAC10280
JAC10290
JAC10300
JAC10310
JAC10320
JAC10330
JAC10340
JAC10350
HEAT0010
HEAT0020
HEAT0030
HEAT0040
HEAT0050
HEAT0060
HEAT0070
HEAT0080
HEAT0090
HEAT0100
HEAT0110
HEAT0120
HEAT0130
HEAT0140
HEAT0150
HEAT0160
HEAT0170
HEAT0180
HEAT0190
HEAT0200
HEAT0210
HEAT0220
HEAT0230
HEAT0240
HEAT0250
HEAT0260
HEAT0270
HEAT0280
HEAT0290
HEAT0300
HEAT0310
HEAT0320
HEAT0330
HEAT0340
HEAT0350
HEAT0360
HEAT0370
HEAT0380
HEAT0390
HEAT0400
HEAT0410
HEAT0420
HEAT0430
HEAT0440
HEAT0450
HEAT0460
HEAT0470

```

C	CALCULATE GRASHOF NUMBER	HEAT0480
C		HEAT0490
	$GRX=(32.00 * DENR) / VISNY2$	HEAT0500
C		HEAT0510
C	CALCULATE NUSSELT NUMBER	HEAT0520
C		HEAT0530
	$CNUX=HWX / CONDX$	HEAT0540
C		HEAT0550
C	CALCULATE RATIO OF NUSSELT NO. AND GRASHOF NO.	HEAT0560
C		HEAT0570
	$RNUGR=-TPWX$	HEAT0580
	$RATIC=CNUX / (GRX / 4.00) ** 0.25$	HEAT0590
C		HEAT0600
C	CALCULATE PRANDTL NUMBER	HEAT0610
C		HEAT0620
	20 $PRIN=(VISC(JJ) * CPH(JJ) * (2.4200)) / COND(JJ)$	HEAT0630
	30 $PR#=(VISC(J) * CPH(J) * (2.4200)) / COND(J)$	HEAT0640
C		HEAT0650
	RETURN	HEAT0660
	END	HEAT0670

REFRIGERANT 114 -- CONSTANT PROPERTY PROBLEM WITH PR = 3.55

NUMBER OF EQUATIONS 5

FINAL NUMBER OF MESH POINTS 97

LEFT BOUNDARY POINT = 0.0 RIGHT BOUNDARY POINT = 10.00

BOUNDARY CONDITIONS

0.0 0.0 0.0 1.00 0.0

MATRIX OF LEFT BOUNDARY CONDITIONS

1.00	0.0	0.0	0.0	0.0
0.0	1.00	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	1.00	0.0
0.0	0.0	0.0	0.0	0.0

MATRIX OF RIGHT BOUNDARY CONDITIONS

0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0	1.00	0.0	0.0	0.0
0.0	0.0	0.0	0.0	0.0
0.0	0.0	0.0	1.00	0.0

TOLERANCE 0.100-05

ERROR ESTIMATE FOR COMPONENTS

0.705D-06 8.120D-06 0.441D-07 0.285D-07 0.298D-07

JERROR = 0

REFRIGERANT 114 -- CONSTANT PROPERTY PROBLEM WITH PR = 3.55

N	ETA	F	FP	FPP	T	TP
1	0.0	0.0	0.41359D-27	0.51455D+00	0.10000D+01	-0.85914D+00
6	0.52083D+00	0.48949D-01	0.15324D+00	0.11462D+00	0.56392D+00	-0.77755D+00
11	0.10417D+01	0.13426D+00	0.15954D+00	-0.57693D-01	0.23313D+00	-0.46395D+00
16	0.15625D+01	0.20714D+00	0.11784D+00	-0.87784D-01	0.70552D-01	-0.14021D+00
21	0.20833D+01	0.25724D+00	0.76286D-01	-0.69767D-01	0.16761D-01	-0.49274D-01
26	0.26042D+01	0.28876D+00	0.46817D-01	-0.45124D-01	0.33795D-02	-0.10740D-01
31	0.31250D+01	0.30799D+00	0.28129D-01	-0.27812D-01	0.61444D-03	-0.10514D-02
36	0.36458D+01	0.31973D+00	0.16755D-01	-0.16747D-01	0.10487D-03	-0.36046D-03
41	0.41667D+01	0.32513D+00	0.99390D-02	-0.99920D-02	0.17231D-04	-0.50188D-04
46	0.46875D+01	0.33015D+00	0.58409D-02	-0.59375D-02	0.27676D-05	-0.67532D-05
51	0.52083D+01	0.33254D+00	0.34720D-02	-0.35213D-02	0.43858D-06	-0.15542D-05
56	0.57292D+01	0.33394D+00	0.20441D-02	-0.20861D-02	0.68952D-07	-0.24530D-06
61	0.62500D+01	0.33477D+00	0.11984D-02	-0.12352D-02	0.10790D-07	-0.18458D-07
66	0.67708D+01	0.33525D+00	0.69772D-03	-0.73112D-03	0.16837D-08	-0.60062D-08
71	0.72917D+01	0.33553D+00	0.40142D-03	-0.43267D-03	0.26232D-09	-0.97671D-09
76	0.78125D+01	0.33569D+00	0.22607D-03	-0.25603D-03	0.40420D-10	-0.14540D-09
81	0.83333D+01	0.33578D+00	0.12232D-03	-0.15149D-03	0.63371D-11	-0.22679D-10
86	0.88542D+01	0.33582D+00	0.60922D-04	-0.89638D-04	0.97179D-12	-0.35310D-11
91	0.93750D+01	0.33585D+00	0.24597D-04	-0.53038D-04	0.13722D-12	-0.54917D-12
96	0.98958D+01	0.33585D+00	0.31033D-05	-0.31342D-04	0.74281D-14	-0.85412D-13

REFRIGERANT 114 -- CONSTANT PROPERTY PROBLEM WITH PR = 3.55

WALL TEMPERATURE = 779.69 R

FREE STREAM TEMPERATURE = 774.69 R

TEMPERATURE DIFFERENCE = 5.00 R

REFERENCE TEMPERATURE = 774.69 R

WALL PRANDTL NUMBER = 2.55

FREE STREAM PRANDTL NUMBER = 3.55

MUSSELT NUMBER = 0.100220+04

GRASHOF NUMBER = 0.736130+14

Q\*X\*\*(1/4) = HEAT TRANSFER IN BTU/HR-FT\*\*(7/4)

H\*X\*\*(1/4) = COEFFICIENT OF HEAT TRANSFER IN BTU/HP-FT\*\*(7/4)-R

NU/GR = RATIO OF MUSSELT NO. AND GRASHOF NO.

Q*X**(1/4)	H*X**(1/4)	NU/GR
0.659863120+02	0.131972620+02	0.859144460+00

## VITA<sup>2</sup>

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Candidate for the Degree of

Doctor of Philosophy

Thesis: LAMINAR FREE CONVECTIVE HEAT TRANSFER IN THE SUPERCRITICAL REGION WITH VARIABLE FLUID PROPERTIES

Major Field: Mechanical Engineering

### Biographical:

Personal Data: Born in Tehran, Iran, June 10, 1951, the son of Iraj and Parvin Jahanshahi Ghajar.

Education: Graduated from Hadaf High School, Tehran, Iran, May, 1969; received the Bachelor of Science degree in Mechanical Engineering from Oklahoma State University, Stillwater, Oklahoma, May, 1974; received the Master of Science degree in Mechanical Engineering from Oklahoma State University, December, 1975; completed requirements for the Doctor of Philosophy degree at Oklahoma State University, July, 1979.

Professional Experience: Graduate Teaching and Research Assistant, School of Mechanical and Aerospace Engineering, Oklahoma State University, August, 1974, to December, 1975; Graduate Teaching and Research Associate, School of Mechanical and Aerospace Engineering, Oklahoma State University, January, 1976, to July, 1979.

Professional Organizations: Associate Member, American Society of Mechanical Engineers (ASME); American Institute of Aeronautics and Astronautics (AIAA); American Society of Heating, Refrigeration and Air-Conditioning Engineers (ASHRAE).

Honors and Awards: Phi Eta Sigma; Pi Tau Sigma; Dean's Honor Roll of Distinguished Students, Oklahoma State University, September, 1970, to May, 1974; recipient of National Iranian Oil Company Scholarship (NIOC), February, 1972, to May, 1974; two-time recipient of International Student Scholarship, Oklahoma State University, September, 1971, to May, 1972, and September, 1973, to May, 1974.