SYNTHESIS OF STEPHENSON TYPE II SIX-LINK

•

FUNCTION GENERATOR FOR FINITELY AND

INFINITESIMALLY SEPARATED

POSITIONS

By

WILLIAM RICHARD COUTANT, JR.

Bachelor of Science

Oklahoma State University

Stillwater, Oklahoma

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Thesis Approved:

Thesis Adviser

n.m. Namoun

Dean of the Graduate College

PREFACE

The purpose of this study was the development of a method for synthesizing the Stephenson Type II six-link function generator for both finite and infinitesimally separated positions. That is, both the positions of the input and output link and the velocity, acceleration, jerk and kerk ratios of the input to the output link are possible design parameters. The procedure developed was incorporated into computer programs for ease of synthesis.

I wish to express my appreciation to my thesis adviser, Dr. A. H. Soni, for his encouragement, understanding and expertise in the field of mechanisms. Special thanks go to Dilip Kohli for his valuable advice. The committee members, Dr. M. Mamoun and Dr. Larry D. Zirkle, deserve recognition for giving both time and suggestions.

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CHAPTER I

INTRODUCTION

One of the reasons for the existence of the field of mechanisms is the need for production of non-linear motion. Because the function generating mechanism can produce non-linear motion, it is a vital tool for industry. Therefore, procedures have been developed for synthesizing the planar six-link function generator.

McLarnan (1)* discussed the procedure for synthesizing six-link mechanisms with complex numbers and numerical techniques for from 6 to 11 finite points. Soni, Varma and Juneja (2) synthesized the six-link mechanism for three finitely separated positions, while Myklebust and Tesar (3) synthesized for five positions. Kaufman (4) synthesized for five positions, correlating coupler motion with input crank rotations. Kim, Hamid and Soni (5) synthesized the six-link mechanism for point path generation. Soni and others (6) made use of the matrix method of synthesis and synthesized for eleven precision positions by numerical methods.

The studies mentioned above indicate there is a need for a clear method for synthesizing the six-link function generator for all combinations of finitely and infinitesimally separated positions when one is

*Numbers in parentheses refer to numbered references in the Bibliography.

obtaining a closed form solution. The matrix method set forth by Suh and Radcliffe (7) proves to be concise and easily adaptable in going from finitely separated position synthesis to infinitesimally separated position synthesis. It was chosen for this thesis work because the equations used for finitely separated position synthesis can be differentiated to obtain infinitesimally separated position synthesis of any degree with minimal change in synthesis procedure.

One may question why the six-link mechanism is practical, having two more linkages and three more revolute pairs than the four-link mechanism. After all, these add to the cost and maintenance involved for industrial practicality. But when one looks at the six-link mechanism's advantages of extra mobility, versatility and compactness over the four-link mechanism, it can be seen that the six-link mechanism is a necessary tool for industry.

All the six-link mechanisms with one degree of freedom come from two kinematic chains. These are shown in Figure 1. These two chains result in five six-link mechanisms, shown in Figure 2.





Figure 1. Six-Link Kinematic Chains



Figure 2. Five Six-Link Mechanisms

As can be seen from Figure 2, the Stephenson Type III mechanism and the Watt Type II mechanism have a basic four-link structure which limits their function generating capabilities to that of a four-link mechanism. The Watt Type I and Stephenson Type I mechanisms have been studied for all possible types of function generation problems. This is because they consist of the four-link mechanism with two binary links connected in series to them. The Stephenson Type I mechanism has the two binary links connected to the coupler point of the four-link mechanism, reducing the function generation problem to that of combining the coupler point motion of the four-link mechanism to the placement and length of the two binary links. For the case of the Watt Type I mechanism the desired function generation is obtained by connecting the output of the first four-link mechanism to the input of the second four-link mechanism. The Stephenson Type II mechanism has the same function generation flexibility as the Stephenson Type I and the Watt Type I mechanisms, but it has the advantage of having only two grounded revolute pairs, thus being applicable to a variety of situations in which three grounded revolute pairs would be impractical or unable to meet design criteria. However, a generalized method of synthesis for the Stephenson Type II mechanism has not been developed.

This thesis involves developing a closed form solution for the synthesis of the Stephenson Type II six-link mechanism function generator for five positions. The matrix approach to synthesis set forth by Suh and Radcliffe (7) and later developed by Kohli and Soni (8) will be used along with the principle of inversion. In order to specify the velocity, acceleration, jerk and kerk (time derivative of jerk) ratios of output link to input link, the principle of infinitesimally separated positions introduced by Mueller (9) and developed by Bottema (10) and Tesar (11, 12, 13) was used.

This thesis will set forth a method of synthesizing the Stephenson Type II six-link mechanism for five precision positions of function generation. Five precision positions were chosen because five is the maximum number of precision positions for which the synthesis equations

can be made linear. From six to a maximum of eleven precision positions the synthesis equations are non-linear and have to be solved by numerical techniques. This method can have the problem of convergence, and the resulting mechanism is the closest solution--not necessarily the exact solution. For three precision positions the design equations are linear and can be solved directly, while for four and five positions the principle of linear superposition, discussed by Mohan Rao and Sandor (14) must be utilized in order to make the equations linear. Five precision positions obviously give the designer more flexibility and are necessary when considering the additional design specifications of velocity, acceleration, jerk and kerk ratios.

Tesar (13) uses a set of nomenclature for describing the possible combinations of finitely separated and infinitesimally separated displacements for five positions. All the possible combinations are:

✓ **P-P-P-P** [∨] P-PP-P-P PP-P-P-P P-P-PP-P P-P-PP ~ PP-PP-P PP-P-PP P-PP-PP v p - p p p - pV PPP-P-P P-P-PPP ~ PPP-PP ~ PP-PPP PPPP-P P-PPPP ✓ PPPPP

Dashes indicate finitely separated points and no dash indicates infinitesimally separated points. This thesis presents an analytical method of synthesizing for all these motions. These combinations can be used to obtain a wide variety of function generation motions.

The synthesis procedure used in this thesis involves, first, using the principle of inversion to transform the synthesis problem into a rigid body guidance four-link mechanism synthesis problem. The matrix approach is used to design a closed form solution for the rigid body guidance problem. The result is the designed six-link mechanism. The

same procedure is then used to synthesize the Stephenson Type II sixlink mechanism for infinitesimally separated positions.

One motivation behind this thesis has been to give industry a simple method for using the six-link mechanism in specific design situations. To do this, computer programs were written to give solutions for any one of the above motion programs.

CHAPTER II

MATRIX METHOD OF SYNTHESIS FOR FINITELY SEPARATED POSITIONS

Mechanisms in one form or another have been used for many hundreds of years to obtain mechanical motion. As engineering and mathematics advanced, it became desirable to develop mathematical methods to synthesize mechanisms to perform desired motions. The computer opened up a whole new field of possible synthesis methods.

One of these methods was set forth by Suh (7), whose paper presents a method for using a generalized displacement matrix to describe rigid body motion. A rigid body is specified when a point on the rigid body is known in a specified coordinate system, and when the angle of rotation of the rigid body with respect to the point is known. As the rigid body executes planar motion these two values are specified as in Figure 3. The points in the rigid body are designated as C_1 , C_2 , and C_3 , and the rotation of the rigid body between positions C_1 and C_2 is α_{12} , and between C_1 and C_3 is α_{13} . Point B_n is any point on the rigid body. The positions of B_2 and B_3 are described by multiplying the generalized displacement matrix times the first position matrix of B_1 .

 $[b_n] = [D_{1n}] [b_1]$ (2-1)

The generalized displacement matrix is given by



Figure 3. Description of Rigid Body Motion

According to the Burmester Theory for three and four finite positions of the rigid body there are an infinite number of circle points (designated as B_n) on the rigid body which follow a circular path. For five finite positions of the rigid body there are a maximum of four

possible points on the rigid body that lie in a circle. A link with one pivot connected to the point B_n on the rigid body and another pivot connected to the fixed pivot A can be described by the equation of a circle, where (X_A, Y_A) are coordinates of fixed pivot A.

$$(X_{bn} - X_{A})^{2} + (Y_{bn} - Y_{A})^{2} = (X_{b1} - X_{A})^{2} + (Y_{b1} - Y_{A})^{2}$$
(2-3)

The design equation is obtained by the substitution of Equation (2-1) into Equation (2-3).

For three finitely separated positions there are two design equations that are linear if any two of the four variables X_A , Y_A , X_{B1} and Y_{B1} are assumed. For four finitely separated positions there are three design equations; therefore, one of the four variables must be assumed. For five finitely separated positions there are four design equations and all four variables are to be determined. In the last two cases the equations are non-linear but can be made linear by applying the principle of linear superposition.

CHAPTER III

MATRIX METHOD OF SYNTHESIS FOR INFINITESIMALLY SEPARATED POSITIONS

In the previous chapter the matrix method of synthesizing a fourlink mechanism for finitely separated positions of point C on the rigid body was discussed. The matrix method can also be used to synthesize the four-link mechanism for infinitesimally separated positions of point C on the rigid body. In order to utilize this method, a new matrix has been developed by Soni (15) which, when multiplied by a finite position of C, results in the infinitesimally separated position of C. To do this, point C on the rigid body and point B, a circle point, are described by vectors as in Figure 4.





Since $\overline{R} = \overline{D} + \overline{r}$, the velocity of point B is given by

$$\frac{d\bar{R}}{dt} = \frac{d\bar{D}}{dt} + \frac{d\alpha}{dt} \bar{K} \times \bar{r}$$
(3-1)

where

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$$\bar{\mathbf{r}} = (\mathbf{X}_{B} - \mathbf{X}_{C}) \ \bar{\mathbf{i}} + (\mathbf{Y}_{B} - \mathbf{Y}_{C}) \ \bar{\mathbf{j}}$$

$$\bar{\mathbf{D}} = \mathbf{X}_{C} \bar{\mathbf{i}} + \mathbf{Y}_{C} \bar{\mathbf{j}}$$

$$\bar{\mathbf{R}} = \mathbf{X}_{B} \bar{\mathbf{i}} + \mathbf{Y}_{B} \bar{\mathbf{j}} .$$
(3-2)

Substituting Equations (3-2) into Equation (3-1) results in

$$\frac{dX_B}{dt}\vec{i} + \frac{dY_B}{dt}\vec{j} = \frac{dX_C}{dt}\vec{i} + \frac{dY_C}{dt}\vec{j} + \vec{K}\frac{d\alpha}{dt} \times [(X_B - X_C)\vec{i} + (Y_B - Y_C)\vec{j}]. \quad (3-3)$$

By changing the independent parameter from t to α Equation (3-3) becomes

$$\frac{\mathrm{d}\mathbf{X}_{\mathrm{B}}}{\mathrm{d}\alpha}\,\overline{\mathbf{i}}\,+\frac{\mathrm{d}\mathbf{Y}_{\mathrm{B}}}{\mathrm{d}\alpha}\,\overline{\mathbf{j}}\,=\frac{\mathrm{d}\mathbf{X}_{\mathrm{C}}}{\mathrm{d}\alpha}\,\overline{\mathbf{i}}\,+\frac{\mathrm{d}\mathbf{Y}_{\mathrm{C}}}{\mathrm{d}\alpha}\,\overline{\mathbf{j}}\,+\,(\mathbf{X}_{\mathrm{B}}-\mathbf{X}_{\mathrm{C}})\,\overline{\mathbf{j}}\,-\,(\mathbf{Y}_{\mathrm{B}}-\mathbf{Y}_{\mathrm{C}})\,\overline{\mathbf{i}}\,. \tag{3-4}$$

By separating the i and j components Equation (3-4) becomes

$$\frac{dX_B}{d\alpha} = \frac{dX_C}{d\alpha} - (Y_B - Y_C)$$

$$\frac{dY_B}{d\alpha} = \frac{dY_C}{d\alpha} + (X_B - X_C).$$
(3-5)

For the nth position, in matrix form this becomes

$$\begin{bmatrix} \dot{x}_{Bn} \\ \dot{y}_{Bn} \\ 1 \end{bmatrix} = \begin{bmatrix} 0 & -1 & \frac{dX_{Cn}}{d\alpha_{1n}} + Y_{Cn} \\ 1 & 0 & \frac{dY_{Cn}}{d\alpha_{1n}} - X_{Cn} \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} X_{Bn} \\ Y_{Bn} \\ 1 \end{bmatrix}$$
(3-6)

In order to develop the first infinitesimally separated position synthesis equation, Equations (2-1) and (3-6) must be substituted into the derivative of Equation (2-3).

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$$(X_{Bn} - X_A) X_{Bn} + 2 (Y_{Bn} - Y_A) Y_{Bn} = 0$$
 (3-7)

In order to find higher orders of infinitesimally separated positions, Equations (3-5) should be differentiated successively. The second order infinitesimally separated position is

$$\frac{d^{2}X_{B}}{d\alpha^{2}} = \frac{d^{2}X_{C}}{d\alpha^{2}} + \frac{dY_{C}}{d\alpha} - \frac{dY_{B}}{d\alpha}$$

$$\frac{d^{2}Y_{B}}{d\alpha^{2}} = \frac{d^{2}Y_{C}}{d\alpha^{2}} + \frac{dX_{B}}{d\alpha} - \frac{dX_{C}}{d\alpha} .$$
(3-8)

Substituting Equations (3-5) into Equations (3-8) results in

$$\frac{d^2 X_B}{d\alpha^2} = \frac{d^2 X_C}{d\alpha^2} + X_C - X_B$$

$$\frac{d^2 Y_B}{d\alpha^2} = \frac{d^2 Y_C}{d\alpha^2} + Y_C - Y_B .$$
(3-9)

The matrix equation is

$$\begin{bmatrix} \mathbf{x} \\ \mathbf{x} \\ \mathbf{Bn} \\ \mathbf{x} \\ \mathbf{y} \\ \mathbf{Bn} \\ \mathbf{1} \end{bmatrix} = \begin{bmatrix} -1 & 0 & \frac{d^{2}\mathbf{x}}{d\alpha_{1n}^{2}} + \mathbf{x}_{Cn} \\ 0 & -1 & \frac{d^{2}\mathbf{y}}{d\alpha_{1n}^{2}} + \mathbf{y}_{Cn} \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \mathbf{x} \\ \mathbf{x} \\ \mathbf{y} \\ \mathbf{Bn} \\ \mathbf{1} \end{bmatrix}$$
(3-10)

Following the same procedure, the third order infinitesimally separated position matrix equation is

$$\begin{bmatrix} x_{Bn} \\ x_{Bn} \\ y_{Bn} \\ 1 \end{bmatrix} = \begin{bmatrix} 0 & 1 & \frac{d^{3}X_{Cn}}{d\alpha_{1n}^{3}} - Y_{Cn} \\ -1 & 0 & \frac{d^{3}Y_{Cn}}{d\alpha_{1n}^{3}} + X_{Cn} \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_{Bn} \\ y_{Bn} \\ 1 \end{bmatrix}$$
(3-11)

and the fourth order infinitesimally separated position matrix is

$$\begin{bmatrix} x_{Bn} \\ x_{Bn} \\ y_{Bn} \\ 1 \end{bmatrix} = \begin{bmatrix} 1 & 0 & \frac{d^{4}X_{Cn}}{d\alpha_{1n}^{4}} - X_{Cn} \\ 0 & 1 & \frac{d^{4}Y_{Cn}}{d\alpha_{1n}^{4}} - Y_{Cn} \\ 0 & 0 & 1 & \end{bmatrix} \begin{bmatrix} x_{Bn} \\ y_{Bn} \\ 1 \end{bmatrix} (3-12)$$

The third and fourth infinitesimally separated position design equations are obtained by substituting the appropriate matrix equation into the derivatives of Equation (3-7).

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CHAPTER IV

SYNTHESIS PROCEDURE FOR FIVE FINITELY SEPARATED POSITIONS

For clarity's sake, a set of nomenclature will be established. The Stephenson Type II six-link mechanism is of the form shown in Figure 5, and will have revolute pairs at A, B, C, D, and E and grounded revolute pairs at M and Q. The input link will be the ternary link MAD while the output will be the link QC. Changes in rotation of link 1 from its first position will be designated as θ_{1n} while changes in rotation with respect to the first position of the output will be Φ_{1n} . Also, the rotation displacements of the ternary link about point C with respect to the first position will be α_{1n} .

To produce a specified function generation the synthesis procedure was changed to a rigid body guidance problem by means of inverting the mechanism. Inversion was taken about ternary link 1. Holding link 1 fixed, the once grounded link MQ was rotated a minus θ_{in} direction for each position of desired input rotation. At the same time, link QC was rotated at an angle $(\Phi_{1n}-\theta_{1n})$ for each position (Figure 6). The displacement equation which gives the positions of X_{C} , Y_{C} in terms of $-\theta_{1n}$ and $(\Phi_{1n}-\theta_{1n})$ is

$$\begin{split} \mathbf{X}_{\text{Cn}} &= (\mathbf{X}_{\text{Cl}} - \mathbf{X}_{\text{Ql}}) \cos (\Phi_{\text{ln}} - \theta_{\text{ln}}) - (\mathbf{Y}_{\text{Cl}} - \mathbf{Y}_{\text{Ql}}) \sin (\Phi_{\text{ln}} - \theta_{\text{ln}}) + \\ & \mathbf{X}_{\text{Ql}} \cos (-\theta_{\text{ln}}) - \mathbf{Y}_{\text{Ql}} \sin (-\theta_{\text{ln}}) \end{split}$$

$$Y_{Cn} = (X_{C1} - X_{Q1}) \sin (\Phi_{1n} - \theta_{1n}) + (Y_{C1} - Y_{Q1}) \cos (\Phi_{1n} - \theta_{1n}) + X_{Q1} \sin (-\theta_{1n}) + Y_{Q1} \cos (-\theta_{1n}).$$
(4-1)



Figure 5. Stephenson Type II Six-Link Mechanism

In order to define the positions of X_{C}^{-} , Y_{C}^{-} in the inverted positions, the length of QC must be assumed in relation to the unit length of MQ, and the initial angle Φ_{1}^{-} must be given. These two parameters are totally arbitrary and can be varied to obtain the optimum mechanism for a desired function, or can be specified to meet additional design criteria.

With the values of $X_{C}^{}$, $Y_{C}^{}$ in the inverted positions known, the synthesis procedure now becomes a four-link rigid body guidance problem. By using the matrix approach described in Chapter II, a matrix

was obtained which describes the positions of points B and E of the ternary link as the rigid body passes through the points X_{C} , Y_{C} and rotates through the inverted displacement angles of α_{ln} .



Figure 6. Inversion About Link 1

The values of α_{ln} that the ternary link is to rotate are also assumed values and can be used to the designer's advantage when synthesizing the six-link function generator. That is, if the designer has a problem in which the ternary link rotations have to be specified, this can be done; or if it is totally arbitrary, its variations will produce different six-link mechanisms. As explained in Chapter I, the four-link rigid body guidance synthesis equation (4-2) is non-linear when five positions are desired.

$$x_{A1} [c_{1n}] + y_{A1} [c_{2n}] + x_{B1} [c_{3n}] + y_{B1} [c_{4n}] = [1 - \cos\alpha_{1n}]$$

$$[x_{A1}x_{B1} + y_{A1}y_{B1}] + \sin\alpha_{1n} [x_{A1}y_{B1} - x_{B1}y_{A1}] + \frac{1}{2}[c_{1n}^{2} + c_{2n}^{2}]$$

$$(4-2)$$

where

$$C_{1n} = X_{Cn} - X_{C1} \cos \alpha_{1n} + Y_{C1} \sin \alpha_{1n}$$

$$C_{2n} = Y_{Cn} - Y_{C1} \cos \alpha_{1n} - X_{C1} \sin \alpha_{1n}$$

$$C_{3n} = X_{C1} - X_{Cn} \cos \alpha_{1n} - Y_{Cn} \sin \alpha_{1n}$$

$$C_{4n} = Y_{C1} - Y_{Cn} \cos \alpha_{1n} + X_{Cn} \sin \alpha_{1n}.$$

To solve this equation in closed form for five positions, the principle of linear superposition must be used. Letting

$$\lambda_{1} = X_{A1} X_{B1} + Y_{A1} Y_{B1} \text{ and}$$

$$\lambda_{2} = X_{A1} Y_{B1} - X_{B1} Y_{A1}$$
(4-3)

Equation (4-2) can be divided into three linear equations (4-4) which can be solved for five positions simultaneously.

$$r_{1}[c_{1n}] + r_{2}[c_{2n}] + r_{3}[c_{3n}] + r_{4}[c_{4n}] = \frac{1}{2}[c_{1n}^{2} + c_{2n}^{2}]$$

$$P_{1}[c_{1n}] + P_{2}[c_{2n}] + P_{3}[c_{3n}] + P_{4}[c_{4n}] = 1 - \cos\alpha_{1n} \qquad (4-4)$$

$$q_{1}[c_{1n}] + q_{2}[c_{2n}] + q_{3}[c_{3n}] + q_{4}[c_{4n}] = \sin\alpha_{1n}$$

$$n = 2, 3, 4, 5$$

where

$$X_{A1} = r_{1} + \lambda_{1} P_{1} + \lambda_{2} q_{1}$$

$$Y_{A1} = r_{2} + \lambda_{1} P_{2} + \lambda_{2} q_{2}$$

$$X_{B1} = r_{3} + \lambda_{1} P_{3} + \lambda_{2} q_{3}$$

$$Y_{B1} = r_{4} + \lambda_{1} P_{4} + \lambda_{2} q_{4}.$$
(4-5)

Substituting Equations (4-5) into the compatibility conditions given by Equations (4-3) will result in

$$F_{1} \lambda_{2}^{2} + (F_{2} \lambda_{1} + F_{3}) \lambda_{2} + F_{4} \lambda_{1}^{2} + F_{5} \lambda_{1} + F_{6} = 0$$

$$G_{1} \lambda_{2}^{2} + (G_{2} \lambda_{1} + G_{3}) \lambda_{2} + G_{4} \lambda_{1}^{2} + G_{5} \lambda_{1} + G_{6} = 0$$
(4-6)

where

$$F_{1} = q_{1} q_{3} + q_{2} q_{4}$$

$$F_{2} = p_{1} q_{3} + q_{1} p_{3} + p_{2} q_{4} + q_{2} p_{4}$$

$$F_{3} = r_{1} q_{3} + q_{1} r_{3} + q_{2} r_{4} + q_{4} r_{2}$$

$$F_{4} = p_{1} p_{3} + p_{2} p_{4}$$

$$F_{5} = p_{1} r_{3} + p_{3} r_{1} + p_{2} r_{4} + p_{4} r_{2} - 1$$

$$F_{6} = r_{1} r_{3} + r_{2} r_{4}$$

$$(4-7)$$

$$G_{1} = q_{1} q_{4} - q_{2} q_{3}$$

$$G_{2} = p_{1} q_{4} + q_{1} p_{4} - p_{2} q_{3} - q_{2} p_{3}$$

$$G_{3} = r_{1} q_{4} + q_{1} r_{4} - r_{2} q_{3} - q_{2} r_{3} - 1$$

$$G_{4} = p_{1} p_{4} - p_{2} p_{3}$$

$$G_{5} = p_{1} r_{4} + r_{1} p_{4} - p_{2} r_{3} - p_{3} r_{2}$$

$$G_{6} = r_{1} r_{4} - r_{2} r_{3}.$$

Using the Sylvester technique results in a fourth order polynomial, Equation (4-8), the roots of which are the values of λ_1 .

$$L_1 \lambda_1^4 + L_2 \lambda_1^3 + L_3 \lambda_1^2 + L_4 \lambda_1 + L_5 = 0$$
 (4-8)

where

$$L_{1} = G_{4}^{2} F_{1}^{2} + G_{2}^{2} F_{1}F_{4} + G_{1}G_{4}F_{2}^{2} + G_{1}^{2} F_{4}^{2} - G_{2}G_{4}F_{1}F_{2} - G_{1}G_{2}F_{2}F_{4} - 2 G_{1}G_{4}F_{1}F_{4}$$

$$L_{2} = 2 G_{4}G_{5}F_{1}^{2} + G_{2}^{2}F_{1}F_{5} + 2 G_{2}G_{3}F_{1}F_{4} - G_{2}G_{5}F_{1}F_{2} - G_{3}G_{4}F_{1}F_{2} - G_{2}G_{4}F_{1}F_{4} + G_{1}G_{5}F_{2}^{2} + 2 G_{1}G_{4}F_{2}F_{3} + 2 G_{1}^{2}F_{4}F_{5} - G_{1}G_{2}F_{2}F_{5} - G_{1}G_{3}F_{2}F_{4} - G_{1}G_{2}F_{3}F_{4} - 2 G_{1}G_{5}F_{1}F_{4} - 2 G_{1}G_{4}F_{1}F_{5}$$

$$L_{3} = 2 G_{4}G_{6}F_{1}^{2} + G_{5}^{2}F_{1}^{2} + G_{2}^{2}F_{1}F_{6} + 2 G_{2}G_{3}F_{1}F_{5} + G_{3}^{2}F_{1}F_{4} - G_{2}G_{6}F_{1}F_{2} - G_{3}G_{5}F_{1}F_{2} - G_{2}G_{5}F_{1}F_{3} - G_{3}G_{4}F_{1}F_{3} + G_{1}G_{6}F_{2}^{2} + 2 G_{1}G_{5}F_{2}F_{3} + G_{1}G_{4}F_{3}^{2} + 2 G_{1}^{2}F_{4}F_{6} + G_{1}^{2}F_{5}^{2} - G_{1}G_{2}F_{2}F_{6} - G_{1}G_{3}F_{2}F_{5} - G_{1}G_{2}F_{3}F_{5} - G_{1}G_{3}F_{3}F_{4} - 2 G_{1}G_{6}F_{1}F_{4} - 2 G_{1}G_{5}F_{1}F_{5} - 2 G_{1}G_{4}F_{1}F_{6}$$

$$(4-9)$$

$$L_{4} = 2 G_{5}G_{6}F_{1}^{2} + 2 G_{2}G_{3}F_{1}F_{6} + G_{3}^{2}F_{1}F_{5} - G_{3}G_{6}F_{1}F_{2} - G_{2}G_{6}F_{1}F_{3} - G_{3}G_{5}F_{1}F_{3} + 2 G_{1}G_{6}F_{2}F_{3} + G_{1}G_{5}F_{3}^{2} + 2 G_{1}^{2}F_{5}F_{6} - G_{1}G_{3}F_{2}F_{6} - G_{1}G_{3}F_{2}F_{6} - G_{1}G_{3}F_{3}F_{5} - 2 G_{1}G_{5}F_{1}F_{6} - 2 G_{1}G_{6}F_{1}F_{5}$$

$$L_{5} = F_{1}^{2} G_{6}^{2} + G_{3}^{2} F_{1}F_{6} - G_{3}G_{6}F_{1}F_{3} + G_{1}G_{6}F_{3}^{2} + G_{1}^{2}F_{6}^{2} - G_{1}G_{3}F_{3}F_{6} - 2 G_{1}G_{6}F_{1}F_{6}$$

The λ_1 roots are substituted into Equations (4-6), to yield roots for λ_2 . With these two roots known, using Equations (4-5), the values of X_{A1} , Y_{A1} , X_{B1} and Y_{B1} can be found.

Since there are a maximum of four possible pairs of λ_1 and λ_2 , there are a maximum of four possible solutions for the center point and

$$L_{1} = G_{4}^{2} F_{1}^{2} + G_{2}^{2} F_{1}F_{4} + G_{1}G_{4}F_{2}^{2} + G_{1}^{2} F_{4}^{2} - G_{2}G_{4}F_{1}F_{2} - G_{1}G_{2}F_{2}F_{4} - 2 G_{1}G_{4}F_{1}F_{4}$$

$$L_{2} = 2 G_{4}G_{5}F_{1}^{2} + G_{2}^{2}F_{1}F_{5} + 2 G_{2}G_{3}F_{1}F_{4} - G_{2}G_{5}F_{1}F_{2} - G_{3}G_{4}F_{1}F_{2} - G_{2}G_{4}F_{1}F_{4} + G_{1}G_{5}F_{2}^{2} + 2 G_{1}G_{4}F_{2}F_{3} + 2 G_{1}^{2}F_{4}F_{5} - G_{1}G_{2}F_{2}F_{5} - G_{1}G_{3}F_{2}F_{4} - G_{1}G_{2}F_{3}F_{4} - 2 G_{1}G_{5}F_{1}F_{4} - 2 G_{1}G_{4}F_{1}F_{5}$$

$$L_{3} = 2 G_{4}G_{6}F_{1}^{2} + G_{5}^{2}F_{1}^{2} + G_{2}^{2}F_{1}F_{6} + 2 G_{2}G_{3}F_{1}F_{5} + G_{3}^{2}F_{1}F_{4} - G_{2}G_{6}F_{1}F_{2} - G_{3}G_{5}F_{1}F_{2} - G_{2}G_{5}F_{1}F_{3} - G_{3}G_{4}F_{1}F_{3} + G_{1}G_{6}F_{2}^{2} + 2 G_{1}G_{5}F_{2}F_{3} + G_{1}G_{4}F_{3}^{2} + 2 G_{1}^{2}F_{4}F_{6} + G_{1}^{2}F_{5}^{2} - G_{1}G_{2}F_{2}F_{6} - G_{1}G_{3}F_{2}F_{5} - G_{1}G_{2}F_{3}F_{5} - G_{1}G_{3}F_{3}F_{4} - 2 G_{1}G_{6}F_{1}F_{4} - 2 G_{1}G_{5}F_{1}F_{5} - 2 G_{1}G_{4}F_{1}F_{6}$$

$$(4-9)$$

$$L_{4} = 2 G_{5}G_{6}F_{1}^{2} + 2 G_{2}G_{3}F_{1}F_{6} + G_{3}^{2}F_{1}F_{5} - G_{3}G_{6}F_{1}F_{2} - G_{2}G_{6}F_{1}F_{3} - G_{3}G_{5}F_{1}F_{3} + 2 G_{1}G_{6}F_{2}F_{3} + G_{1}G_{5}F_{3}^{2} + 2 G_{1}^{2}F_{5}F_{6} - G_{1}G_{3}F_{2}F_{6} - G_{1}G_{3}F_{2}F_{6} - G_{1}G_{3}F_{3}F_{5} - 2 G_{1}G_{5}F_{1}F_{6} - 2 G_{1}G_{6}F_{1}F_{5}$$

$$L_{5} = F_{1}^{2} G_{6}^{2} + G_{3}^{2} F_{1}F_{6} - G_{3}G_{6}F_{1}F_{3} + G_{1}G_{6}F_{3}^{2} + G_{1}^{2}F_{6}^{2} - G_{1}G_{3}F_{3}F_{6} - 2 G_{1}G_{6}F_{1}F_{6}$$

The λ_1 roots are substituted into Equations (4-6), to yield roots for λ_2 . With these two roots known, using Equations (4-5), the values of X_{A1} , Y_{A1} , X_{B1} and Y_{B1} can be found.

Since there are a maximum of four possible pairs of λ_1 and λ_2 , there are a maximum of four possible solutions for the center point and the circle point. The number of solutions depends on whether λ_1 and λ_2 have imaginary roots. Only real roots for λ_1 and λ_2 result in possible solutions. A choice of any two of these center point and circle point combinations will result in a maximum of six solutions for a designed four-link mechanism.

TABLE I

FIVE FINITE POSITIONS

INPUT ROTATION ANGLES ARE	-15.000	-25.000	-40.000	-60.000
OUTPUT ROTATION ANGLES ARE	-20.000	-30.000	-50.000	-75.000
TERNARY LINK ROTATIONS ARE	10.000	20.000	30.000	40.000
OUTPUT LINK IS 1.0 LONG	AND AT AN	ANGLE OF	100.0 DEGREES	

SOLUTION

Y

Х

М	0.0000000	0.00000000
Q	1.00000000	0.00000000
C1	0.82635313	0.98480803

POSSIBLE SOLUTION FOR POINTS A1 AND B1 OR D1 AND E1

CENTER	R POINT	CIRCLE	E POINT		
Х	Y	x	Y		
0.03711897	0.09393525	0.21974057	0.34299600		
-5.62751389	-8.39265823	-8.16039371	-5.00552368		
ROOT IS IMAG	INARY				

ROOT IS IMAGINARY

Once this is done, the six-link mechanism is designed in its first position. The center points of the four-link mechanism are now points A and D of the input ternary link, and the corresponding circle points are the points B and E of the moving ternary link. Now by grounding link MQ the desired input-output relationship is obtained. A computer program that follows this procedure is listed in Appendix A, and an example problem is presented in Table I. The resulting six-link mechanism will go through the five desired input and output rotations, while the ternary link goes through a total rotation of $(\alpha_{1n} + \theta_{1n})$. The ternary link rotation is now relative to grounded link MQ.

CHAPTER V

SYNTHESIS PROCEDURE FOR FIRST INFINITESIMALLY SEPARATED POSITIONS

A continuation of the procedure set forth in Chapter IV is the development of a method that will specify the velocity ratio of the input to output at a certain finite position of the input and output links. In Chapter IV it was shown that the synthesis for five positions involved, first, inversion and then four-link rigid body guidance synthesis. The same procedure will be followed for synthesizing the Stephenson Type II six-link mechanism for velocity ratios. The displacement of the output link is a function of the input link, and both are a function of time; that is, $\Phi(t) = f(\theta(t))$.

In order to obtain a specified velocity ratio, this relationship must be differentiated with respect to time, resulting in $\frac{d\Phi}{dt} = \frac{d\Phi}{d\theta} \frac{d\theta}{dt}$. It can be seen that the infinitesimal displacement $\frac{d\Phi}{d\theta}$, can be specified by the ratio of $\frac{d\Phi}{dt}$ to $\frac{d\theta}{dt}$, both of which are design parameters. For the function $\Phi = f(\theta)$, as shown in Figure 7, the infinitesimal displacement is the direction and magnitude of the slope at a point. In other words, the desired velocity ratios are now converted into an infinitesimal displacement problem.

For infinitesimal displacement for five positions, there are two basic combinations.

PP-P-P-P	PP-PP-P
P-PP-P-P	PP-P-PP
P-P-PP-P	P-PP-PP
P-P-P-PP	

That is, the two combinations are specifying four finite positions and an infinitesimal displacement at one of the four finite positions, or specifying three finite positions and an infinitesimal displacement at two of the finite positions.



Displacement

The principle of inversion is still valid for this case. The four finite positions can be defined by the procedure set forth in Chapter IV. To find the infinitesimal displacement of point C, the displacement Equation (4-1) will be differentiated with respect to θ , resulting

$$\frac{dX_{Cn}}{d\theta_{1n}} = (X_{C1} - X_{Q1}) [\sin (\Phi_{1n} - \theta_{1n}) - \sin (\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}}] + (Y_{C1} - Y_{Q1}) [\cos (\Phi_{1n} - \theta_{1n}) - \cos (\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}}] + X_{Q1} \sin(-\theta_{1n}) + Y_{Q1} \cos (-\theta_{1n})
$$\frac{dY_{Cn}}{d\theta_{1n}} = (X_{C1} - X_{Q1}) [-\cos (\Phi_{1n} - \theta_{1n}) + \cos (\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}}] + (Y_{C1} - Y_{Q1}) [\sin (\Phi_{1n} - \theta_{1n}) - \sin (\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}}] - X_{Q1} \cos (-\theta_{1n}) + Y_{Q1} \sin(-\theta_{1n}).$$
(5-1)$$

This results in a value for $\frac{dX_{Cn}}{d\theta_{1n}}$, $\frac{dY_{Cn}}{d\theta_{1n}}$ which is the infinitesimal displacement of point C with respect to θ_{1n} at a certain point. As explained in Chapter III, infinitesimal displacements for rigid body guidance can be obtained by means of the matrix method. The infinitesimal displacement of point C in designing the four-link rigid body guidance mechanism is defined by rotations α_{1n} . In other words, the infinitesimal displacement of point C given by $\frac{dX_{Cn}}{d\alpha_{1n}}$, $\frac{dY_{Cn}}{d\alpha_{1n}}$ is necessary for the four-link synthesis. Since

$$\frac{dX_{Cn}}{d\alpha_{1n}} = \frac{d\theta_{1n}}{d\alpha_{1n}} \frac{dX_{Cn}}{d\theta_{1n}} \text{ and } \frac{dY_{Cn}}{d\alpha_{1n}} = \frac{d\theta_{1n}}{d\alpha_{1n}} \frac{dY_{Cn}}{d\theta_{1n}}$$
(5-2)

the infinitesimal movement of $\frac{d\theta_{ln}}{d\alpha_{ln}}$ must be given. This is arbitrary and can be used to the designer's advantage.

In order to obtain the synthesis equations for synthesizing the four-link mechanism, the principle set forth in Chapter III may be utilized. The matrix Equations (2-1) and (3-6) are substituted into Equation (3-7) to obtain the design Equation (5-3).

$$\begin{split} x_{A1} & \left(\frac{dx}{d\alpha_{1n}} + x_{C1} \sin\alpha_{1n} + y_{C1} \cos\alpha_{1n}\right) + \\ y_{A1} & \left(\frac{dy}{d\alpha_{1n}} + y_{C1} \sin\alpha_{1n} - x_{C1} \cos\alpha_{1n}\right) + \\ x_{B1} & \left[-\left(\frac{dx}{d\alpha_{1n}} + y_{Cn}\right) \cos\alpha_{1n} - \left(\frac{dy}{d\alpha_{1n}} - x_{Cn}\right) \sin\alpha_{1n}\right] + \\ y_{B1} & \left[-\left(\frac{dy}{d\alpha_{1n}} - x_{Cn}\right) \cos\alpha_{1n} + \left(\frac{dx}{d\alpha_{1n}} + y_{Cn}\right) \sin\alpha_{1n}\right] = \\ \sin\alpha_{1n} & \left(x_{A1}x_{B1} + y_{A1}y_{B1}\right) + \cos\alpha_{1n} & \left(x_{A1}y_{B1} - x_{B1}y_{A1}\right) - \\ \cos\alpha_{1n} & \left(x_{C1} \frac{dx_{Cn}}{d\alpha_{1n}} + y_{C1} \frac{dy_{Cn}}{d\alpha_{1n}} - y_{C1}x_{Cn} + x_{C1}y_{Cn}\right) + \\ \sin\alpha_{1n} & \left(y_{C1} \frac{dx_{Cn}}{d\alpha_{1n}} - x_{C1} \frac{dy_{Cn}}{d\alpha_{1n}} + x_{C1}x_{Cn} + y_{C1}y_{Cn}\right) + \\ x_{Cn} & \frac{dx_{Cn}}{d\alpha_{1n}} + y_{Cn} \frac{dy_{Cn}}{d\alpha_{1n}} - \\ x_{Cn} & \frac{dx_{Cn}}{d\alpha_{1n}} + y_{Cn} \frac{dy_{Cn}}{d\alpha_{1n}} + \\ x_{Cn} & \frac{dx_{Cn}}{d\alpha_{1n}} + y_{Cn} \frac{dy_{Cn}}{d\alpha_{1n}} + \\ x_{Cn} & \frac{dx_{Cn}}{d\alpha_{1n}} + x_{Cn} \frac{dy_{Cn}}{d\alpha_{1n}} + \\ x_{Cn} & \frac{dx_{Cn}}{d\alpha_{1n}} + \\ x_$$

Please note that at this point a quicker and easier procedure for obtaining this equation may be followed. This procedure will be of particular use when developing higher order infinitesimal displacement synthesis equations. For infinitesimal movement, the changes in the positions of B and E are very small. For displacement analysis the equation of a circle is used in defining the movement of circle point B around center point A. Equation (2-1) is substituted into the equation of a circle, which results in the following equation.

$$f(X_{B1}, Y_{B1}, X_{A1}, Y_{A1}, \alpha_{1n}) = 0$$
 (5-4)

In order to obtain the infinitesimal movement, this equation must be differentiated. By means of the Taylor series, neglecting higher order terms, the derivative of this function can be given by Equation (5-5).

$$f(X_{B1}, Y_{B1}, X_{A1}, X_{A1}, \alpha_{1n}) + \frac{\partial f(X_{B1}, Y_{B1}, X_{A1}, Y_{A1}, \alpha_{1n})}{\partial \alpha_{1n}} \Delta \alpha_{1n} + \dots = 0$$
(5-5)

The first part has been satisfied by the equation (5-4) and the second part has yet to be satisfied. Therefore, in order to obtain the infinitesimal displacement equation for synthesis, the finite position synthesis Equation (4-2) must be differentiated with respect to α_{1n} . Therefore, the first infinitesimal displacement synthesis equation may be obtained by differentiating the coefficients of the finite displacement synthesis Equation (4-2).

Since Equation (5-3) is of the same form as Equation (4-2), the linear superposition principle allows for Equation (4-2) to be substituted where desired. The four positions result in three displacement equations which become linear after applying the principle of linear superposition. One more equation is needed to solve for the four unknowns. The infinitesimal displacement equation is also linear and has the same unknowns, therefore it is the desired fourth equation. The method used in solving for the four unknowns will be identical to that used for solving the five finite displacement problem in Chapter IV.

A computer program has been written which results in the synthesized six-link Stephenson Type II mechanism for four finite positions and a specified velocity ratio at one of the finite positions. It is listed in Appendix B and an example is presented in Table II. A computer program has also been written for the case of three finite positions and a velocity ratio specified at any two of the finite positions. It is listed in Appendix C, and an example is presented in Table III.

TABLE II

FOUR FINITE POSITIONS AND ONE VELOCITY RATIO (VELOCITY RATIO IS SPECIFIED AT POINT 4)

INPUT VELOCITY = 1.0	OUTPUT	VELOCITY = 2.0	C
INPUT ROTATION ANGLES ARE	-15,000	-30.000	-45.000
OUTPUT ROTATION ANGLES ARE	-20.000	-45.000	-75.000
TERNARY LINK ROTATIONS ARE	10.000	20.000	30.000

OUTPUT LINK IS 1.0 LONG AND AT AN ANGLE OF 100.0 DEGREES.

FIRST INFINITESIMAL DISPLACEMENTS

DPHDTH(J) = 2.0DTHDAL(J) = 1.0

SOLUTION

Х		Y		
М	0.0000000	0.0000000		
Q	1.00000000	0.0000000		
C1	0.82635210	0.98480770		

POSSIBLE SOLUTION FOR POINTS A1 AND B1 OR D1 AND E1

	CENTER	POINT	CIRCLE POINT
x		Y	x

X	Y	Х	Y
-0.22540400	1.59108100	-0.25595210	1.53780500
-0.48618850	1.69824500	-0.40762280	1.74143700

ROOT IS IMAGINARY

ROOT IS IMAGINARY

TABLE III

THREE FINITE POSITIONS AND TWO VELOCITY RATIOS (VELOCITIES SPECIFIED AT POINTS 1 AND 3)

1 3	INPUT INPUT	VELOCI VELOCI	TY = 1.0 TY = 1.0)				0 0	UTPUI UTPUI	VE VE	LOCIT LOCIT	Y = Y =	0.0 0.0
	IN OU' TEI	PUT R(TPUT H RNARY	DTATION A ROTATION LINK ROT	NGLES ANGLES ATIONS	ARE ARE ARE		-15.0 -20.0 10.0	000 000 000	-	30. 60. 20.	000 000 000		
	OUTPUT	LINK	IS 1.0 I	ONG AN	D AT	AN	ANGLE	OF	-30.0	DE	GREES	•	
	FIRST	INFI	VITESIMAI	. DISPL	ACEME	ENTS	3	DP DT	HDTH (HDAL (]) :]) :	= 0.0 = 1.0		
	FIRST	INFI	NITES IMAI	. DISPL	ACEME	ENTS	5	DP DT	HDTH (HDAL (K) K) =	= 0.0 = 2.0		
				<u>50</u>	LUTIC	<u>ON</u>							
			x				Y	Y					
		М	0.0000	000			0.0000	0000	0				

Q	1.0000000	0.0000000
C1	1.86602400	-0.49999999

POSSIBLE SOLUTION FO S A1 AND B1 OR D1 AND E1

Y

-0.72231820

-1.81822200

-2.69249400

-1.16569700

Y

-0.74633180

-1.85000600

-2.82720700

-1.61335900

CENT

X

1.38239300

1.34985100

1.94092400

4.05168700

TER	PO	INT	
-----	----	-----	--

POIN	Т
------	---

CIRCLE POINT

Х

1.58071100

1.25711400

1.79815800

4.92417600
CHAPTER VI

SYNTHESIS PROCEDURE FOR SECOND INFINITESIMALLY SEPARATED POSITIONS

In addition to the velocity ratio at a finite point, another design parameter might be to specify an acceleration ratio for the input and output at the same finite point. The input acceleration is the second derivative of θ with respect to time, while the output acceleration is the second derivative of Φ with respect to time. This ratio can be obtained by differentiating $\Phi(t) = f(\theta(t))$ twice. The result is

$$\frac{d^2\Phi}{dt^2} = \frac{d^2\Phi}{d\theta^2} \left(\frac{d\theta}{dt}\right)^2 + \frac{d\Phi}{d\theta} \frac{d^2\theta}{dt^2}.$$
 (6-1)

To find the second infinitesimal displacement $\frac{d^2 \Phi}{d\theta^2}$, all other terms must be specified. For a specific problem, this involves specifying

 $\frac{d^2 \Phi}{dt^2} = \text{the acceleration of the output link,}$ $\frac{d^2 \theta}{dt^2} = \text{the acceleration of the input link,}$ $\frac{d\theta}{dt} = \text{the velocity of the input link, and}$ $\frac{d\Phi}{d\theta} = \text{the first infinitesimal displacement.}$

The term $\frac{d\Phi}{d\theta}$ has already been determined from specifying the velocity ratio at the same finite point.

For five positions the second infinitesimal displacement involves the following class of problems:

PPP-P-P	PPP-PP
P-PPP-P	PP-PPP
P-P-PPP	

That is, the two combinations are specifying three finite points and the first and second infinitesimal displacements at one of the finite points, or specifying two finite points and first and second infinitesimal displacements at one of the finite points and a first infinitesimal displacement at another finite point.

In order to synthesize the Stephenson Type II six-link mechanism for the above two cases, the same procedure employed in the previous chapters will be used. Inversion will be taken about the input link. The links MQ and QC are rotated according to the principle of inversion to find the finite points X_{Cn} , Y_{Cn} . The first and the second derivatives of the displacement equations are obtained by differentiating Equations (4-1) with respect to θ to find $\frac{dX_{Cn}}{d\theta}$, $\frac{d^2X_{Cn}}{d\theta^2}$, and $\frac{d^2Y_{Cn}}{d\theta^2}$. The first derivative Equations (5-1) are given in Chapter V and the second derivative equation is

$$\frac{d^{2}X_{Cn}}{d\theta_{1n}^{2}} = (X_{C1} - X_{Q1}) \left(-\cos \left(\Phi_{1n} - \theta_{1n}\right) + 2\cos \left(\Phi_{1n} - \theta_{1n}\right) \frac{d\Phi_{1n}}{d\theta_{1n}} -\cos \left(\Phi_{1n} - \theta_{1n}\right) \frac{d\Phi_{1n}}{(d\theta_{1n})^{2}} - \sin \left(\Phi_{1n} - \theta_{1n}\right) \frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}}\right) + (Y_{C1} - Y_{Q1}) \left(\sin \left(\Phi_{1n} - \theta_{1n}\right) - 2\sin \left(\Phi_{1n} - \theta_{1n}\right) \frac{d\Phi_{1n}}{d\theta_{1n}} + \sin \left(\Phi_{1n} - \theta_{1n}\right) \frac{d\Phi_{1n}}{(d\theta_{1n})^{2}} - \cos \left(\Phi_{1n} - \theta_{1n}\right) \frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}}\right) - X_{Q1} \cos \left(-\theta_{1n}\right) + Y_{Q1} \sin \left(-\theta_{1n}\right) \qquad (6-2)$$

$$\frac{d^2 Y_{Cn}}{d\theta_{ln}^2} = (X_{Cl} - X_{Ql}) (-\sin (\Phi_{ln} - \theta_{ln}) + 2 \sin (\Phi_{ln} - \theta_{ln}) \frac{d\Phi_{ln}}{d\theta_{ln}} -$$

$$\sin \left(\Phi_{1n}^{-\theta} - \Phi_{1n} \right) \left(\frac{d\Phi_{1n}}{d\theta_{1n}} \right)^{2} + \cos \left(\Phi_{1n}^{-\theta} - \Phi_{1n} \right) \frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}})$$

$$+ \left(Y_{C1}^{-Y} - Y_{Q1} \right) \left(-\cos \left(\Phi_{1n}^{-\theta} - \Phi_{1n} \right) + 2 \cos \left(\Phi_{1n}^{-\theta} - \Phi_{1n} \right) \frac{d\Phi_{1n}}{d\theta_{1n}} \right)$$

$$- \cos \left(\Phi_{1n}^{-\theta} - \Phi_{1n} \right) \left(\frac{d\Phi_{1n}}{d\theta_{1n}} \right)^{2} - \sin \left(\Phi_{1n}^{-\theta} - \Phi_{1n} \right) \frac{d^{2}\Phi_{1n}}{d\theta_{2n}^{2}})$$

$$- X_{O1} \sin \left(-\theta_{1n} \right) - Y_{O1} \cos \left(-\theta_{1n} \right) .$$

With the first and second infinitesimal displacements of X_{Cn} , Y_{Cn} known, the synthesis procedure now involves a four-link rigid body guidance synthesis problem with first and second infinitesimal displacements. Notice that the four-link synthesis procedure involves the first and second infinitesimal displacement of X_{Cn} , Y_{Cn} with respect to the rotation of the ternary link. That is, $\frac{dX_{Cn}}{d\alpha_{1n}}$, $\frac{d^2X_{Cn}}{d\alpha_{1n}^2}$ and $\frac{d^2Y_{Cn}}{d\alpha_{1n}^2}$

are to be specified in synthesizing the four-link mechanism. Equations (5-2) must be differentiated with respect to α_{1n} , resulting in

$$\frac{d^{2}X_{Cn}}{d\alpha_{1n}^{2}} = \frac{d^{2}\theta_{1n}}{d\alpha_{1n}^{2}} \frac{dX_{Cn}}{d\theta_{1n}} + \left(\frac{d\theta_{1n}}{d\alpha_{1n}}\right)^{2} \frac{d^{2}X_{Cn}}{d\theta_{1n}^{2}}$$

$$\frac{d^{2}Y_{Cn}}{d\alpha_{1n}^{2}} = \frac{d^{2}\theta_{1n}}{d\alpha_{1n}^{2}} \frac{dY_{Cn}}{d\theta_{1n}} + \left(\frac{d\theta_{1n}}{d\alpha_{1n}}\right)^{2} \frac{d^{2}Y_{Cn}}{d\theta_{1n}^{2}}.$$
(6-3)

The ratio $\frac{d^2 \Phi_{\ln}}{d \theta_{\ln}^2}$ has to be specified along with $\frac{d \Phi_{\ln}}{d \theta_{\ln}}$, which was needed

for the first infinitesimal displacement synthesis. Again, these are arbitrary values, or can be used to the designer's advantage.

As a result of the justification established in Chapter V, the coefficients of the design Equation (4-2) can be differentiated twice with respect to α_{ln} to obtain the second infinitesimal displacement four-link mechanism synthesis Equation (6-4).

$$cos\alpha_{1n} (A1^{H}B1 + A1^{H}B1) = cos\alpha_{1n} (A1^{H}B1 + B1^{H}B1) = a_{B1}^{H}A1^{H}B1$$

$$cos\alpha_{1n} (2 Y_{C1} \frac{dX_{Cn}}{d\alpha_{1n}} - 2 X_{C1} \frac{dY_{Cn}}{d\alpha_{1n}} - Y_{C1} \frac{d^{2}Y_{Cn}}{d\alpha_{1n}^{2}} - X_{C1} \frac{d^{2}X_{Cn}}{d\alpha_{1n}^{2}} + X_{C1}^{H}X_{Cn} + Y_{C1}^{H}Y_{Cn}) + x_{C1}^{H}X_{Cn} + Y_{C1}^{H}Y_{Cn} + y_$$

$$\sin\alpha_{1n} (2 X_{C1} \frac{dX_{Cn}}{d\alpha_{1n}} + 2 Y_{C1} \frac{dY_{Cn}}{d\alpha_{1n}} - X_{C1} \frac{d^2Y_{Cn}}{d\alpha_{1n}^2} + Y_{C1} \frac{d^2X_{Cn}}{d\alpha_{1n}^2} + Y_{C1} \frac{d^2X_{Cn}}{d\alpha_{1n}^2} - Y_{C1}X_{Cn} + X_{C1}Y_{Cn}) + X_{Cn} \frac{d^2X_{Cn}}{d\alpha_{1n}^2} + Y_{Cn} \frac{d^2Y_{Cn}}{d\alpha_{1n}^2} + \frac{dY_{Cn}}{d\alpha_{1n}^2} + \frac{dY_{Cn}}{d\alpha_{1n}^2} \frac{dY_{Cn}}{d\alpha_{1n}^2} + \frac{dY_{Cn}}{d\alpha_{1n}^2} \frac{dY_{Cn}}{d\alpha_{1n}^2} + \frac{dY_$$

Since this equation is still of the same form as the finite displacement and first infinitesimal displacement equations, it can be used as a fourth equation that is needed due to the reduction of one finite position equation. Two computer programs have been written which cover the two classes of problems discussed in this chapter. A computer program which synthesizes the Stephenson Type II mechanism for three finite positions and first and second infinitesimally separated positions of input to output specified at one of the finite positions is listed in Appendix D, and an example problem is presented in Table IV.

TABLE IV

THREE FINITE POSITIONS AND ONE ACCELERATION RATIO (ACCELERATION RATIO IS AT POINT 3)

INPUT VELOCITY = 1.0 INPUT ACCELERATION = 2.0	OUTPUT VELOCITY = 0.0 OUTPUT ACCELERATION = 0.0
INPUT ROTATION ANGLES ARE OUTPUT ROTATION ANGLES ARE TERNARY LINK ROTATIONS ARE	-25.000 -50.000 -10.000 -30.000 5.000 15.000
OUTPUT LINK IS 1.0 LONG AND AT AN	ANGLE OF 100.0 DEGREES.
FIRST INFINITESIMAL DISPLACEMENTS	DPHDTH(J) = 0.0 DTHDAL(J) = 1.0
SECOND INFINITESIMAL DISPLACEMENT	DDPHTH(J) = 0.0 $DDTHAL(J) = 1.0$
SOLUTION	
Х	Y
M 0.0000000 Q 1.0000000 C1 0.82635313	0.0000000 0.0000000 0.98480803
POSSIBLE SOLUTION FOR POINTS A	1 AND B1 OR D1 AND E1
CENTER POINT	CIRCLE POINT
Х У	Х Ү
-1.47581768 -0.56793118 1.31134033 6.33950806 -1	0.37606907 1.30638123 0.17961121 -15.10237122
ROOT IS IMAGINARY	
ROOT IS IMAGINARY	

A computer program which synthesizes the Stephenson Type II mechanism for two finite positions and first and second infinitesimally separated positions of input to output at one of the finite points, as well as a first infinitesimally separated position specified at the other finite point is listed in Appendix E, and an example problem is presented in Table V.

TABLE V

TWO FINITE POSITIONS, ONE VELOCITY RATIO, AND ONE ACCELERATION RATIO (VELOCITY RATIO IS AT POINT 1 AND ACCELERATION RATIO IS AT POINT 2)

1 2 2	INPUT VELOCIT INPUT VELOCIT INPUT ACCELER	Y = 1.0 Y = 1.0 ATION = 0.0	OUT OUT OUT	PUT VELOCITY = 3.0 PUT VELOCITY = 2.0 PUT ACCELERATION = 0.	0
	IN OU TE	PUT ROTATION AND TPUT ROTATION AN RNARY LINK ROTAT	GLE IS NGLE IS TION IS	30.000 45.000 10.000	
	OUTPUT LINK I	S 1.0 LONG AND A	AT AN ANGLE O	F 60.0 DEGREES.	
	FIRST INFINIT	ESIMAL DISPLACE	IENTS	DPHDTH(K) = 3.0 DTHDAL(K) = 1.0	
	SECOND INFINI	TESIMAL DISPLACE	MENTS	DDPHTH(J) = 0.0 $DDTHAL(J) = 1.0$	
	FIRST INFINIT	ESIMAL DISPLACEN	1ENTS	DPHDTH(J) = 2.0 $DTHDAL(J) = 1.0$	
		SOLU	JTION .		
		X	Y		
	М	0.0000000	0.000	00000	
	Q	1.0000000	0.000	00000	
	C1	1.5000000	0.866	02520	
	POSSIBLE SOL	UTION FOR POINTS	S A1 AND B1 O	R D1 AND E1	
	CENTER	POINT	CIRC	LE POINT	
	X	Y	Х	Y	
	1.74607800	-0.63277810	1.88640900	-0.50722500	
	2.98628100	-1.30977000	3.82497700	-1.56066700	
	ROOT IS IMAGIN	ARY			
	ROOT IS IMAGIN	ARY			

,

CHAPTER VII

SYNTHESIS PROCEDURE FOR THIRD AND FOURTH INFINITESIMALLY SEPARATED POSITIONS

An additional design criterion might be the specification of the jerk and kerk ratios of input to output. For example, at a finite point it may be desirable not only to specify a zero acceleration ratio, but also to have a zero jerk and kerk ratio. The jerk and kerk ratios enable the designer to have greater control over his input-output linkage motion.

The same synthesis procedure set forth in the previous chapters will be used for these next two cases, which are

> PPPP-P PPPPP P-PPPP

Third Infinitesimally Separated Position

In order to solve the third infinitesimal displacement problem, one must realize that at a finite position, the first, second and third infinitesimal displacement parameters must be given. That is, $\frac{d\Phi}{d\theta}$ is first found from the specified velocity ratios and, in turn, is used in Equation (6-1) to find $\frac{d^2\Phi}{d\theta^2}$. Now the derivative of Equation (6-1) can be taken to solve for the third infinitesimally separated position $\frac{d^3\Phi}{d\theta^3}$.

$$\frac{d^{3}\Phi}{dt^{3}} = \frac{d^{3}\Phi}{d\theta^{3}} \left(\frac{d\theta}{dt}\right)^{3} + 3 \frac{d\theta}{dt} \cdot \frac{d^{2}\theta}{dt^{2}} \cdot \frac{d^{2}\Phi}{d\theta^{2}} + \frac{d^{3}\theta}{dt^{3}} \cdot \frac{d\Phi}{d\theta}$$
(7-1)

These values are then used in Equations (5-1), (6-2) and (7-2) to find the values for $\frac{dX_{Cn}}{d\theta_{1n}}$, $\frac{dY_{Cn}}{d\theta_{1n}}$, $\frac{d^2X_{Cn}}{d\theta_{1n}^2}$, $\frac{d^2Y_{Cn}}{d\theta_{1n}^2}$, $\frac{d^3X_{Cn}}{d\theta_{1n}^3}$, and $\frac{d^3Y_{Cn}}{d\theta_{1n}^3}$. $\frac{d^3X_{Cn}}{d\theta_{1n}^3} = (X_{C1} - X_{Q1})$ [-sin $(\Phi_{1n} - \theta_{1n}) + 3 \sin (\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}}$ $- 3 \sin (\Phi_{1n} - \theta_{1n}) (\frac{d\Phi_{1n}}{d\theta_{1n}})^2 + 3 \cos (\Phi_{1n} - \theta_{1n}) \frac{d^2\Phi_{1n}}{d\theta_{1n}^2}$ $- 3 \cos (\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}} \cdot \frac{d^2\Phi_{1n}}{d\theta_{1n}^2} + \sin(\Phi_{1n} - \theta_{1n}) (\frac{d\Phi_{1n}}{d\theta_{1n}})^3$ $- \sin (\Phi_{1n} - \theta_{1n}) \frac{d^3\Phi_{1n}}{d\theta_{1n}^3}] +$

$$(Y_{C1} - X_{Q1}) \left[-\cos(\Phi_{1n} - \theta_{1n}) + 3\cos(\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}} \right]$$

$$- 3\cos(\Phi_{1n} - \theta_{1n}) (\frac{d\Phi_{1n}}{d\theta_{1n}})^{2} - 3\sin(\Phi_{1n} - \theta_{1n}) \frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}}$$

$$+ 3\sin(\Phi_{1n} - \theta_{1n}) \frac{d\Phi_{1n}}{d\theta_{1n}} \cdot \frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}} + \cos(\Phi_{1n} - \theta_{1n}) (\frac{d\Phi_{1n}}{d\theta_{1n}})^{3}$$

$$- \cos(\Phi_{1n} - \theta_{1n}) \frac{d^{3}\Phi_{1n}}{d\theta_{1n}^{3}} - X_{Q1}\sin(-\Phi_{1n}) - Y_{Q1}\cos(-\theta_{1n})$$
(7-2)

$$\begin{aligned} &(Y_{C1}-Y_{Q1})[-\sin(\Phi_{1n}-\theta_{1n})+3\sin(\Phi_{1n}-\theta_{1n})\frac{d\Phi_{1n}}{d\theta_{1n}} \\ &-3\sin(\Phi_{1n}-\theta_{1n})\frac{d\Phi_{1n}}{(d\theta_{1n})}^{2}+3\cos(\Phi_{1n}-\theta_{1n})\frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}} \\ &-3\cos(\Phi_{1n}-\theta_{1n})\frac{d\Phi_{1n}}{d\theta_{1n}}\cdot\frac{d^{2}\Phi_{1n}}{d\theta_{1n}^{2}}+\sin(\Phi_{1n}-\theta_{1n})\frac{d\Phi_{1n}}{d\theta_{1n}})^{3} \\ &-\sin(\Phi_{1n}-\theta_{1n})\frac{d\Phi_{1n}}{d\theta_{1n}^{3}}]+X_{Q1}\cos(-\theta_{1n})-Y_{Q1}\sin(-\theta_{1n}) \end{aligned}$$

These values are, in turn, changed to $\frac{dX_{Cn}}{d\alpha_{1n}}$, $\frac{dY_{Cn}}{d\alpha_{1n}}$, $\frac{d^2X_{Cn}}{d\alpha_{1n}^2}$, $\frac{d^2Y_{Cn}}{d\alpha_{1n}^2}$, $\frac{d^3X_{Cn}}{d\alpha_{1n}^2}$, $\frac{d^3X_{Cn}}{d\alpha_{1n}^2}$

and
$$\frac{d^{3}Y_{Cn}}{d\alpha_{1n}^{3}}$$
 by means of Equations (5-2), (6-3) and (7-3).

$$\frac{d^{3}X_{Cn}}{d\alpha_{1n}^{3}} = \frac{d^{3}\theta_{1n}}{d\alpha_{1n}^{3}} \cdot \frac{dX_{Cn}}{d\theta_{1n}} + 3 \frac{d\theta_{1n}}{d\alpha_{1n}} \cdot \frac{d^{2}\theta_{1n}}{d\alpha_{1n}^{2}} \cdot \frac{d^{2}X_{Cn}}{d\theta_{1n}^{2}} + (\frac{d\theta_{1n}}{d\alpha_{1n}})^{3} \frac{d^{3}X_{Cn}}{d\theta_{1n}^{3}}$$

$$\frac{d^{3}Y_{Cn}}{d\alpha_{1n}^{3}} = \frac{d^{3}\theta_{1n}}{d\alpha_{1n}^{3}} \cdot \frac{dY_{Cn}}{d\theta_{1n}} + 3 \frac{d\theta_{1n}}{d\alpha_{1n}} \cdot \frac{d^{2}\theta_{1n}}{d\alpha_{1n}^{2}} \cdot \frac{d^{2}Y_{Cn}}{d\theta_{1n}^{2}} + (\frac{d\theta_{1n}}{d\alpha_{1n}})^{3} \frac{d^{3}Y_{Cn}}{d\theta_{1n}^{3}}$$
(7-3)

At a finite point, the first, second and third infinitesimal displacement synthesis Equations (5-3), (6-4) and (7-4), respectively, can now be used to design the desired Stephenson Type II motion.

$$\begin{array}{l} X_{A1} & (\frac{d^{3}X_{Cn}}{d\alpha_{1n}^{3}} - X_{C1} \sin\alpha_{1n} - Y_{C1} \cos\alpha_{1n}) + \\ Y_{A1} & (\frac{d^{3}Y_{n}}{d\alpha_{1n}^{3}} + X_{C1} \cos\alpha_{1n} - Y_{C1} \sin\alpha_{1n}) + \\ X_{B1} & [-\frac{d^{3}X_{Cn}}{d\alpha_{1n}^{3}} + 3 \frac{d^{2}Y_{Cn}}{d\alpha_{21n}^{2}} - 3\frac{dX_{Cn}}{d\alpha_{1n}}Y_{Cn}) \cos\alpha_{1n} \\ & - (\frac{d^{3}Y_{Cn}}{d\alpha_{1n}^{3}} - 3 \frac{d2X_{Cn}}{d\alpha_{21n}^{2}} - 3\frac{dY_{Cn}}{d\alpha_{1n}} + X_{Cn}) \sin\alpha_{1n}] + \end{array}$$

$$\begin{split} \mathbf{Y}_{\text{B1}} & \left[- \left(\frac{d^{3}\mathbf{Y}_{\text{Cn}}}{d\alpha_{1n}^{2}} - 3 \frac{d^{2}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} - 3 \frac{d\mathbf{Y}_{\text{Cn}}}{d\alpha_{1n}} + \mathbf{X}_{\text{Cn}} \right) \cos\alpha_{1n} \\ & + \left(\frac{d^{3}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} + 3 \frac{d^{2}\mathbf{Y}_{\text{Cn}}}{d\alpha_{1n}^{2}} - 3 \frac{d\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}} - \mathbf{Y}_{\text{Cn}} \right) \sin\alpha_{1n} \right] = \\ & - \sin\alpha_{1n} \left(\mathbf{X}_{\text{A1}} \mathbf{X}_{\text{B1}} + \mathbf{Y}_{\text{A1}} \mathbf{Y}_{\text{B1}} \right) - \cos\alpha_{1n} \left(\mathbf{X}_{\text{A1}} \mathbf{Y}_{\text{B1}} - \mathbf{X}_{\text{B1}} \mathbf{Y}_{\text{A1}} \right) \\ & + \cos\alpha_{1n} \left(-\mathbf{Y}_{\text{C1}} \frac{d^{3}\mathbf{Y}_{\text{Cn}}}{d\alpha_{1n}^{2}} - \mathbf{X}_{\text{C1}} \frac{d^{3}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} + 3 \mathbf{Y}_{\text{C1}} \frac{d^{2}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} \right] \\ & + 3 \mathbf{X}_{\text{C1}} \frac{d^{3}\mathbf{Y}_{\text{Cn}}}{d\alpha_{1n}^{2}} - \mathbf{X}_{\text{C1}} \frac{d^{3}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} + 3 \mathbf{Y}_{\text{C1}} \frac{d^{2}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} \right] \\ & + 3 \mathbf{X}_{\text{C1}} \frac{d^{3}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} + 3 \mathbf{Y}_{\text{C1}} \frac{d^{3}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} - \mathbf{Y}_{\text{C1}}\mathbf{X}_{\text{Cn}} + \mathbf{X}_{\text{C1}}\mathbf{Y}_{\text{Cn}} \right) \\ & + 3 \mathbf{X}_{\text{C1}} \frac{d^{3}\mathbf{Y}_{\text{Cn}}}{d\alpha_{1n}^{2}} + \mathbf{Y}_{\text{C1}} \frac{d^{3}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} + 3 \mathbf{X}_{\text{C1}} \frac{d^{2}\mathbf{X}_{\text{Cn}}}{d\alpha_{1n}^{2}} + 3 \mathbf{X}_{\text{C1}} \frac{d^{2}\mathbf{X}_{\text{Cn}}}{d\alpha_{1$$

Since the equations are all of the same form, the principle of linear superposition still applies.

A computer program which synthesizes the Stephenson Type II mechanism for two finite positions and first, second and third infinitesimally separated positions of input to output at one of the finite positions is listed in Appendix F, and an example problem is presented in Table VI.

TABLE VI

TWO FINITE POSITIONS AND ONE JERK RATIO (JERK RATIO IS SPECIFIED AT POINT 2)

INPUT VELOCITY = 1.0OUTPUT VELOCITY = -2.5INPUT ACCELERATION = 0.0OUTPUT ACCELERATION = -1.0INPUT JERK = 0.0OUTPUT JERK = 0.0INPUT ROTATION ANGLE IS 30.000 OUTPUT ROTATION ANGLE IS -45.000 TERNARY LINK ROTATION IS -25.000OUTPUT LINK IS 1.2 LONG AND AT AN ANGLE OF -30.0 DEGREES. FIRST INFINITESIMAL DISPLACEMENTS DPHDTH(J) = -2.52 DTHDAL(J) = 1.0SECOND INFINITESIMAL DISPLACEMENTS DDPHTH(J) = -1.0DDTHAL(J) = 1.0THIRD INFINITESIMAL DISPLACEMENTS DDDPTH(J) = 0.0DDDTAL(J) = 1.0SOLUTION х Y М 0.00000000 0.0000000 0.0000000 1.00000000 Q C1 2.03922900 -0.59999970 POSSIBLE SOLUTION FOR POINTS A1 AND B1 OR D1 AND E1 CENTER POINT CIRCLE POINT Х Х Y Y 2.43076600 -0.57500930 1.09700200 -0.47618860 -4,92590300 4.04580400 5.21653500 -3.42272900 ROOT IS IMAGINARY ROOT IS IMAGINARY

TABLE VII

ONE FINITE POSITION AND ONE KERK RATIO

INPUT VELOCITY = 1.0 OUTPUT VELOCITY = 2.0 INPUT ACCELERATION = 0.0OUTPUT ACCELERATION = 3.0OUTPUT JERK = 0.0INPUT JERK = 0.0INPUT KERK = 0.0 OUTPUT KERK = 0.0OUTPUT LINK IS 1.5 LONG AND AT AN ANGLE OF 30.0 DEGREES. FIRST INFINITESIMAL DISPLACEMENTS DPHDTH(J) = 2.0DTHDAL(J) = 1.0SECOND INFINITESIMAL DISPLACEMENTS DDPHTH(J) = 3.0DDTHAL(J) = 1.0THIRD INFINITESIMAL DISPLACEMENTS DDDPTH(J) = 0.0DDDTAL(J) = 1.0FOURTH INFINITESIMAL DISPLACEMENTS DDDDPT(J) = 0.0DDDDTA(J) = 1.0

SOLUTION

	Х	Y
М	0.0000000	0.00000000
Q	1.00000000	0.00000000
C1	2.29903793	0.74999970

POSSIBLE SOLUTIONS FOR POINTS A1 AND B1 OR D1 AND E1

CENTER	POINT	CIRCLE	POINT
X	Y	X	Y
1.58438492	-0.57121086	2.80116749	1.10895443
4.47628021	0.35051918	7.80039597	0.81995583
ROOT IS IMAGI	INARY	•	
ROOT IS IMAGE	INARY		

Fourth Infinitesimally Separated Position

The procedure stated above can now be expanded to include the fourth infinitesimally separated position synthesis problem. The procedure will be the same, except that Equations (7-1), (7-2), (7-3) and (7-4) must be differentiated. Since there is an additional design equation for this case, the first, second, third and fourth infinitesimal displacements can be specified only at one finite point.

A computer program which synthesizes the Stephenson Type II mechanism for one finite point and first, second, third and fourth infinitesimally separated positions of input to output is listed in Appendix G, and an example problem is presented in Table VII.

CHAPTER VIII

SUMMARY AND CONCLUSIONS

This thesis develops an approach for synthesizing the Stephenson Type II six-link function generator for five positions of input and output. These five positions consist of both finitely and infinitesimally separable positions. There are fourteen possible combinations of finite and infinitesimally separated positions for which function generation motion can be obtained.

The synthesis procedure consists of starting with desired finite displacements, velocity ratios, acceleration ratios, jerk ratios or kerk ratios, whichever are desired, and changing them to an nth order infinitesimally separated position synthesis problem. The principle of inversion is used to transform the Stephenson Type II mechanism synthesis problem into a four-link rigid body guidance problem. The matrix method proved to be easily adaptable for synthesizing any one of the fourteen motion programs. In order to obtain the closed form solution for synthesis, the principle of linear superposition was utilized.

Once the synthesis procedure for the Stephenson Type II six-link function generator was developed, seven computer programs were written to perform any of the fourteen function generation motions. Feeding the desired design parameters into the programs will result in from 0 to 6 possible solutions for the six-link mechanism because from 0 to 4

center point and circle point combinations are possible for the rigid body motion.

With these seven basic programs, a wide variety of function generation problems can be solved. For example, the Stephenson Type II mechanism can be synthesized for five finite positions, where the output link goes through more than 180 degrees. If cutting action is desired, ternary link rotations can be specified along with the output link rotations, resulting in the desired relative motion between two connecting rigid bodies. Since it was necessary in the synthesis procedure for nth order infinitesimally separated positions to specify $\frac{d\theta_{\ln}}{d\alpha_{\ln}}$, $\frac{d^2\theta_{\ln}}{d\alpha_{\ln}^2}$, etc., an additional parameter in which the ternary link infinitesimal rotations are related to the input infinitesimal rotations is given the designer. When an object is to be picked up at zero velocity and left off at zero velocity, this can be done for three finite positions by using the program in Appendix C. Or specific forces at a specified velocity and position can be obtained by achieving an appropriate acceleration of the output link. An example might be a stamping action. This can be accomplished, depending upon the other design parameters, by using programs in Appendices D, E, F or G. Another major area in which these programs could be used is finite dwell problems. Any of the programs in Appendices A, B, C, D, E or F can be used to obtain a finite dwell of the output while the input link rotates through finite positions. For five finite positions all five of the output rotations can be set equal to zero, while the input goes through the desired rotation angles. The program in Appendix C can be used to specify one finite position and a dwell at another finite posi-The dwell would be developed by specifying two finite points tion.

and a zero velocity ratio at those two points. The same procedure can also be used for programs in Appendices B, D, E and F by setting the appropriate velocity, acceleration and jerk ratios equal to zero. These programs will, of course, result in an approximate finite dwell, but error can be minimized by determining which program is the best for the specific problem.

The synthesis procedure shown in this thesis can be used for synthesis of different mechanisms, not only for finite positions, but also for infinitesimally separated positions. It can be applied to both function generation synthesis problems and rigid body guidance synthesis problems. This versatile and concise method was ideal for synthesis of the Stephenson Type II six-link function generator, which resulted in a practical vehicle for solving design problems.

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APPENDIX A

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SYNTHESIS PROGRAM FOR FIVE

FINITE POSITIONS

1	2345678901	23456789012345	6789012345	578901234567	890123456789012	3456789012345678
С	*****	****	*****	****	****	*****
Ē	*					
č	*	SYNT	HESTS OF S	TX-I TNK STEP	HENSON 2	
č	*	5111,	EUNCTI	DN GENERATOR	TENSON E	1
č	-		104011	ON CENERALON		
5	*		ETVE ETNI			
. L	*		FINC FINI	IE POSTITONS		
C	*					•
Ç	*					
C	*.	INPUT	FINITE ROL	ATIONS	TH1(I)	
С	* '		•			
¢	*	OU TPUT	FINITE RO	TATIONS	PH1(I)	:
С	*					
с	*	ASSUME	D ROTATION	S OF TERNARY	AL1(I)	:
Ċ	*					· · · ·
č	*	GUT PUT	LINK LENG	TH	90	,
ř	*					
ř		OUTPUT	TTNK TNET	TAL ANGLE	THE	
č	*	001101	CINK INT		1116	
ž		** ** **				د ماید ماید ماید ماید ماید ماید ماید مای
5	****		••••••	* • • • • * * • • • • • • • • •	***********	*****
C						
	DIMEN	SICN XCOP(5),C	UF(5), RUUI	K(4), KUU 1 (4	1	•
	DIMEN	SION XQ(5) YQ(5) XA(5) Y	A(5].,X8(5),Y	B(5),XC(5),YC(5	• 2
	DIMEN	SION TH1(5),PH	1(5),AL1(5)		
	DIMEN	SION H(16),H1(16), H2(16)	,R{4},P(4),G	(4)	
	READ	(5,10) TH1(2),	TH1(3),TH1	(4),TH1(5)		
	READ	(5,10) PH1(2),	PH1(3),PH1	(4),PH1(5)		
	READ	(5.10) AL1(2).	AL1 (3) .AL1	(4).AL1(5)		
	10 FORMA	T (4F10.0)				
	PEAC	(5.20) OC. THE				
	20 500 84	T (2510 0)				
	WR 110	T (11 DEV 201	FAVE FANT	-		
	6 FURMA	1 1// 202,228	FIVE FINIT	E PUSITIONS,	//1	
	WRITE	(6,11) H1(2)	,TH1(3), IH	1(4) + TH1(5)		
	11 FORMA	T (9X, 26H INPU	T ROTATION	ANGLES ARE,	2X,4F10.3,/)	
	WRITE	(6,12) PH1(2)	, PH1 (3), PH	1(4),PH1(5)		
	12 FORMA	T (9X,27H OLTP	UT ROTATIO	N ANGLES ARE	,1X,4F10.3,/)	
	WRITE	(6,13) AL1(2)	,AL1(3),AL	1(4),AL1(5)		
	13 FORMA	T (9X,23H TERN	IARY LINK R	OT AT IONS, 5X,	4F10.3,//}	
	WRITE	(6,14) QC, THE				
	14 FORMA	T (12X,15H OUT	PUT LINK I	S, F10.3, 24H	LONG AND AT AN	ANGLE OF F
	\$10.3.	//)				
	XO(1)	=1.0				
	Y0/11	=0.0				
		0				
		۰. ۱				
	TM=0.	U				
	THE = (1HE#3.14159265	1/ 180.0			
	DO 21	I=2,5				
	TH1(1)=(TH1(I)*3.14	159265)/18	0.0		
	PH1 (I)=(PH1(I)*3.14	1592651/18	0.0		
	21 AL1(I)=(AL1(I)*3.14	159265)/18	0.0		
	XC(1)	=1.0+0C *C0 S(T)	E)			
	VC/11	=0.0+0C*SIN(TH	(F)			
		1-2.5				
		-1-240	T 1 1			
	14(1)	-T *0 42 TIAL _ LUTL				

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C •	T234201940T234201940T234201940T234201940T234201940T234201940T234201840T234201840T234201840T234201840T234201840T
U A	
	$\sum_{i=1}^{n} \sum_{j=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i=1}^{n} \sum_{i$
	$b_1 = b_1 + b_2 + b_1 $
	$S_{2} = S_{2} + S_{2$
	51 31 PUKMAI (12X,3H AL)18X,3H TU,18X,3H XC,18X,3H TC,/)
	$\frac{33}{10} \text{ NRIFE } \{0, 32\} \text{ AUTISTUTISAUTISTUTIS}$
	D4
	$\sum_{i=1}^{n} \frac{1}{1} + \sum_{i=1}^{n} \frac{1}{1} $
	$\frac{1}{10} \qquad 0 \qquad$
	$\frac{1}{2} \qquad \qquad \text{LL SINGLARY TRACK}$
	51 E2-FLITELJIELSIELSFELJSFELZSELELSELSELSELSELSELSELSELSELSELSELSELSE
	72 I J-NI 1/74 J/74 1/7 NJ/74 2/7 NT/74 T/7 NT/7
	73
	74
•	20 (2-KITI-46(4)-46(1)-4(1)-4(1)-4(2)-4(2)-1.0
,	yy
1	╱╻╶────────────────────────────────────
L L	
1	J) ● 0-5 L+5 3+F 3+F 0-2 * 5+5 L+5 1 * 5+5 2 * 5 2
T,	
÷	ער פין יבער יו זיינטיר ו זיינטיר גער איז גער איז גער גער איז גער ער איז גער
1	┙┍ ϶┇┇┲╘╛┲╔╘┲╓╘┙┇┇┲╚┇┲╔┋┲┎┇┱╔╗┲╔┇┯╔┇┯╔┇┲╘╝┲╓╘┇┲╔┇┲╚╝┲╠╋╋╋╄╏╇┠ ╲┑ ἐϗ
ţ	J/ ≠≠ No Yr NE(3)=2 N±C4±C4±E1±E1±C5±C5±E1±E1±C7±C2±C2±C1±E4±2 Λ±C2±C3±C1±E4±C3
1	ijġ

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CARD	
109	\$*G3*F1*F4-G2*G6*F1*F2-G3*G5*F1*F2-G2*G5*F1*F3-G3*G4*F1*F3+G1*G6*F2
110	\$*F2+2.0*G1*G5*F2*F3+G1*G4*F3+F3+2.0*G1*F4*F6+G1*G1*F5*F5=G1*G2*
111	\$F2*F6-G1*G3*F2*F5-G1*G2*F3+F5-G1*G3*F3+F4-2.0*G1*G6*F1*F4-2.0*G1*G
112	\$5*F1*F5-2.0*G1*G4*F1*F6
113	XCDF(4)=2.0+G4+G5+F1+F1+G2+G2+F1+F5+2.0+G2+G3+F1+F4-G2+G5+F1+F2-G3
114	\$*G4*F1*F2-G2*G4*F1*F3+G1*G5*F2*F2+2.0*G1*G4*F2*F3+2.0*G1*G1*F4*F3+
115	\$G1*G2*F2*F5-G1*G3*F2*F4-G1*G2*F3*F4-2.0*G1*G5*F1*F4-2.0*G1*G4*F1*F
116	\$5
117	XCOF(5)=G4*G4*F1*F1+G2*G2*F1*F4-G2*G4*F1*F2+G1*G4*F2*F2+G1*G1*F4*F
118	\$4-G1*G2*F2*F4-2 。 0*G1*G4*F1*F4
119	CALL POLRT(XCOF;COF;4,ROOTR,ROOT1,IER)
120	WRITE (6,109)
121	109 FORMAT (/,27X,9H L1 RGOTS,/)
122	DO 119 $I=1,4$
123	110 WRITE (6,112) RCCTR(I),ROOTI(I)
124	112 FORMAT (17X, E13.5, 10X, E13.5,/)
125	WRITE (6+114) IER
126	114 FORMAT (1X,11)
127	WRITE (6,106)
128	106 FORMAT (1H1,//)
129	125 DO 215 I≑1,4
130	0A=F1
131	08=F2*R00TR(I)+F3
132	DC = F4 * ROOTR(I) * ROOTR(I) + F5 * ROOTR(I) + F6
133	0D=G1
134	DE = G2 * ROO TR (I) + G3
135	0F=G4*R00TR(I)*R00TR(I)+G5*R00TR(I)+G6
136	$IF (ABS(ROOTI(I))) = I_0 OOOOOOI) ROOTI(I) = 0.0$
137	IF(R00TI(II)) 200.130.200
138	200 WRITE (6-201)
130	201 FORMAT (5X-18H REGT IS IMAGINARY./)
140	
141	
142	IF (SA) 200-140
142	$140 \text{ point } A_1 = (-0.8 + S0.8 T (SA))/(2.0 + 1A)$
144	ROTA2 = (-OR - SORT(SA))/(2 - O + OA)
145	
146	JE (SB) 200-150-150
147	150 PD(18) = (-0.5 + 500 T(58))/(2 - 0.800)
320	
140	WRITE (6,155) RCCTA1, RCCTA2, ROOTR1, ROOTR2
150	155 500 MAT 1/-132,84 000 A1 == 518.6.52,84 000 A2 == 518.6.7.127.84 000 B1
150	to the first of the state in the state of th
121	
152	
100	
124	
100	
120	
151	160 WRITE (0,1007 RUDIAL
128	I COLLEGATION SECTION RULE IS \$F20.2777
159	
160	
161	I (U MKI IE 10, IGU) KULIAZ
162	

	000000001111111111122222222233333333334444444445555555556666666666
	123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890
CARD	
163	210 WRITE (6+212)
164	212 FORMAT (14X,13H CENTER POINT,31X,13H CIRCLE POINT,/,8X,3H XA,19X,3
165	\$H YA,19X,3H XB,15X,3H YB,/}
166	XA(I)=R(1)+P(1)*R00TR(I)+Q(1)*R00T
167	YA(1)=R(2)+P(2)+RCOTR(1)+Q(2)+ROOT
168	XB(I)=R(3)+P(3)*RDDTR(I)+Q(3)*ROGT
169	Y B(I)=R(4)+P(4)+ROOTR(I)+Q(4)+ROOT
170	WRITE (6,230) XA(I),YA(I),XB(I),YB(I)
171	230 FORMAT (4F20.0,/,33x,20H XXXXXXXXXXXXXXXXX,//)
172	215 CONTINUE
173	STOP
174	END

APPENDIX B

SYNTHESIS PROGRAM FOR FOUR FINITE POSITIONS

AND ONE FIRST INFINITESIMAL

DISPLACEMENT

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	12345	678901234567890123456789012345678901234567890	123456789012345678901234567890
CARD	C + + + +	**************************************	***
2	C *	אילי שני אלי אילי שני שני שני שני אילי אילי אילי אילי אילי אילי שני אילי אילי אילי אילי אילי אילי אילי אי	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~
3	C *	SYNTHESIS OF SIX-LINK STEPHEN	SON 2 *
4	C *	FUNCTION GENERATOR	*
5	C *		*
6	C *	FOUR FINITE POSITIONS AND ONE VELOC	ITY RATIO *
7	C *		*
8	C *	THOUT FINITE DOT/ TIONS	* Tup (T) +
, 9	t * د ش	INPUT FINITE RUTATIONS	(111(1) *
11	ст С ж	OUTPLE FINITE ROTATIONS	РН1(Т) *
12	C *		*
13	Ç*	ASSUMED ROTATIONS OF TERNARY	AL1(I) *
14	C *		*
15	C *	FIRST INFINITESIMAL DISPLACEMENT	DPHDTH(J) *
16	C *		*
17	C *	OUTPUT LINK LENGTH	QC *
18	C *	OUT DUT A TAKE TAXT TAL ANCE F	* TUE +
19	ι. * c.*	UUTPUT LINK INITIAL ANGLE	1mc *
20	C ***	*****	******
22	c	•	
23	•	DIMENSION XCOF(5),COF(5),ROOTR(4),ROOTI(4)	
24		DIMENSION XQ(5), YQ(5), XA(5), YA(5), XB(5), YB(5),XC(5),YC(5)
25		DIMENSIGN THL(5),PHL(5),ALL(5)	
26		DIMENSION $H(16)$, $H1(16)$, $H2(16)$, $R(4)$, $P(4)$, $Q(4)$	· · ·
27		DIMENSION DXC(5), DYC(5), PXC(5), PYC(5)	
28		DIMENSION DPHDIH(D), DIHDAL(D)	
29	2	KCAD (D)D/J FORMAT (11)	
31	2	WRITE (6.6) .1	
32	6	FORMAT (//. 16X. 28H VELOCITY RATIO IS SPECIFI	ED AT POINT .11.//)
33		READ (5,10) TH1 (2), TH1 (3), TH1 (4)	
34		READ (5,10) PH1(2),PH1(3),PH1(4)	·
35		READ (5,10) AL1(2), AL1(3), AL1(4)	•
36	10	FORMAT (3F10.0)	
37		READ (5,20) QC, THE	
38	20	READ (3420) UPHDIMLJJADIHUAL(J) Endmat (2810 0)	
29	20	WRITE (A.11) TH1(2), TH1(3), TH1(4)	
41	11	FORMAT (9X.26H INPUT ROTATION ANGLES ARE.2X.	3F10.3./)
42		WRITE (6,12) PH1(2), PH1(3), PH1(4)	
43	12	FORMAT (9X, 27H OUTPUT ROTATION ANGLES ARE, 1X	,3F10.3,/)
44		WRITE (6,13) AL1(2), AL1(3), AL1(4)	
45	13	FORMAT (9X,23H TERNARY LINK ROTATIONS,5X,3F1	0.3,//)
46		WRITE (6,14) QC,THE	
47	14	FORMAT (9X,15H CUTPUT LINK IS,FLO.3,24H LONG	AND AT AN ANGLE OF 1 F1
48		30.07777 WRITE (6.15) DRETH(1),DTHDAL(1)	· .
47 7 5 A	15	EORMAT (9X.11H DPHDTH($.1$) = F10.3.10X.11H DTHD	AL(J)=+F10.3.//)
51	10	XQ(1)=1.0	
52		YQ(1)=0.0	
53		XM= 0.0	
54		YM=0.0	

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CARD	
55	TH1(1)=0.0
56	PH1(1)=0.0
57	AL1(1)=0.0
58	THE=(THE≭3.14155265)/180.0
59	DO 21 I=2,4
60	TH1(I)=(TH1(I)*3.14159265)/180.0
61	PH1(I) = (PH1(I) * 3.14159265) / 180.0
62	21 $AL1(I) = (AL1(I) * 3.14159265) / 180.0$
63	XC(1)=1.0+QC*COS(THE)
64	YC(1)=0.0+QC*SIN(THE)
65	DO 30 1=2.4
66	XQ(T)=1-0*CQS(-TH1(T))
67	$Y_{0}(I) = 1 - 0 + S_{1}(I)$
68	XC(1) = XC(1) + COS(PH1(1) - TH1(1)) - YC(1) + SIN(PH1(1) - TH1(1)) + XO(1) - XO(1)
69	<pre>\$)*COS(PH1(I)-TH1(I))+YQ(1)*SIN(PH1(I)-TH1(I))</pre>
70	30 YC(1)=XC(1)*SIN(PH1(1)-TH1(1)}+YC(1)*CDS(PH1(1)-TH1(1))+YC(1)-XC(1)
71	$s_1 + s_1 + (1) - T + (1) + y_0(1) + C + (1) - T + (1)$
72	DXC(J) = (XC(J) - XQ(J)) * (SIN(PH)(J) - TH)(J) - SIN(PH)(J) - TH (J) + PPHOTH
73	(1) + (YC(1) - YO(1)) + (CCS(2H)(1) - TH(1)) - COS(2H)(1) - TH(1)) + CPHDTH(1)
74	(1) + (1)
75	DYC(1) = (XC(1)) + XC(1)) + (-COS(PH)(1)) + TH(1)) + COS(PH)(1) - TH(1)) + DPHOT
76	(+) $(+)$
77	(1) - (0, 1) + (0,
70	
70	$\frac{1}{12} = \frac{1}{12} $
80	$\frac{1}{2} \frac{1}{2} \frac{1}$
21	
01	
02	TELEVELLE VELLE VO. 107 30 VO. 107 30 VC. 107 30 VC. /1
60	DO 23 T-1 4
04	
02	$33 \text{ WRITE } \{0, 52\} AUTI/AUTI/AUTI/AUTI/AUTI/AUTI/AUTI/AUTI/$
00	52 FURMAL (0A)FT2+057A5 12+057A5FT2+057A5FT2+077A5FT2+0777
01	
00	$40 \Pi(I - A(I) + I) - A(I) + I = 0 (I + I) + I = 0 (I + I) = 0 $
07	
90	
91	
92	P(G) =
93	
94	$OU = \Pi(1 \neq 0) - \Pi(1 \neq 0) = \Pi(1 \neq 1) + \Pi(1 \neq 1 \neq 1 \neq 1) = \Pi(1 \neq 1) + \Pi(1 \neq 1$
95	$\Pi(12) = - FAC(3) + CS(ALT(3)) - FTC(3) + STR(ALT(3)) + STR(ALT(3)) - TC(3)$
96	
97	
98	(0 + (1 + 12) = -A(1 + 4) + COS(ALL(1 + 1)) + A(1 + 5) + A(1 + 1))
99	H(18)=PAC(3)+SIN(ALI(3))=PIC(3)+CUS(ALI(3)
100	▶) # SIN(ALI(J)) >>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>
101	
102	
103	
104	
105	$\begin{array}{c} 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 \\ 0 $
106	80 Kilj=0.57 (Nilj=F(1])=N(1+4)=F(1)=0.000 (N=VC/1)=DVC/(N=VC/1)=0.000 (N=VC/1)=0.000 (N=VC/1)=0
107	$R[4] = \{Y[L]\} = A[C]] = A[C]] = Y[C]] = Y[C] = Y[Y[C]] = A[C]] = X[C]] = X[C$
108	\$)}+{XC(1}#XU(J}+YC(L)#YC(J)=XC(1)#PYC(J)+YC(1)#PXC(J)}*SIN(AL1(J))

109	(t)3Y9*(t)3X9*(t)3X9*
110	20.90 I = 1 • 3
111	90 $P(I) = 1 - 0 - C \Pi S (\Delta I (I + 1))$
112	P(4) = STN(A 1(1))
112	
114	100 (11-1)
114	
115	
116	CALE SIMU(H), K, 4, KS)
117	
118	CALL SIMQ(H2,Q,4,KS2)
119	WRITE (6,99) KS
120	WRITE (6,99) KS1
121	WRITE (6,99) KS2
122	99 FORMAT (1X, 11)
123	F1=Q(1)*Q(3)+Q(2)*Q(4)
124	F2=P(1)*Q(3)+Q(1)*P(3)+P(2)*Q(4)+Q(2)*P(4)
125	F3=R(1)*Q(3)+Q(1)*R(3)+Q(2)*R(4)+Q(4)*R(2)
126	F4=P(1)*P(3)+P(2)*P(4)
127	F5=P(1)*R(3)+P(3)*R(1)+P(2}*R(4)+P(4)*R(2)-1.0
128	F6=R(1)*R(3)+R(2)*R(4)
129	G1=Q(1)*Q(4)-Q(2)*Q(3)
130	G2=P(1)*Q(4)+Q(1)*P(4)-P(2)*Q(3)-Q(2)*P(3)
131	G3=R(1)*Q(4)+Q(1)*R(4)-R(2)*Q(3)-Q(2)*R(3)-1.0
132	G4=P(1)*P(4)-P(2)*P(3)
133	G5=P(1)*R(4)+R(1)*P(4)-P(2)*R(3)-P(3)*R(2)
134	G6=R(1)*R(4)-R(2)*R(3)
135	XCDF(1)=F1*F1*G6*G6+G3*G3*F1*F6-G3*G6*F1*F3+G1*G6*F3*F3+G1*G1*F6*F
136	\$6-G1*G3*F3*F6-2•0*G1*G6*F1*F6
137	XCOF(2)=2.0*G5*G6*F1*F1+2.0*G2*G3*F1*F6+G3*G3*F1*F5-G3*G6*F1*F2-G2
138	\$*G6*F1*F3-G3*G5*F1*F3+2.0*G1*G6*F2*F3+G1*G5*F3*F3+2.0*G1*G1*F5*F6+
139	\$G1 *G3 *F2 *F6 -G1 *C2 *F3 *F6 -G1 *G3 *F3 *F5 - 2.0 *G1 *G5 *F1 *F6 - 2.0 *G1 *G6 *F1 *F
140	\$5
141	XCDF(3)=2.0*G4*C6*F1*F1+G5*G5*F1*F1+G2*G2*F1*F6+2.0*G2*G3*F1*F5+G3
142	\$*G 3# F1* F4+G2*G6*F1*F2-G3*G5*F1*F2-G2*G5*F1*F3-G3*G4*F1*F3+G1*G6*F2
143	\$*F2+2,0*G1*G5*F2*F3+61*G4*F3*F3+2,0*G1*G1*F4*F6+61*G1*F5*F5-61*G2*
144	\$E2*E6-G1*G3*E2*E5-G1*G2*E3*E5-G1*G3*E3*E4-2.0*G1*G6*E1*E4-2.0*G1*G
145	\$5*F1*F5-2_0*G1*G4*F1*F6
146	$Y(n \in I \land i = 2)$. 0 × (0 × (0 × (1 × (1 × (0 × (0 × (0 ×
147	\$\$ 64 \$ F1 \$ F2 = 62 \$ 64 \$ F1 \$ F3 + 63 \$ 65 \$ F2 \$ 75 \$ 7 \$ 63 \$ 63 \$ F2 \$ 53 \$ 13 \$ 7 \$ 63 \$ 63 \$ 63 \$ 63 \$ 63 \$ 63 \$ 63
149	
140	
150	¥CDE(5)=C4*C4*F1*F1+C2*C2*F1*F4-C2*C4*F1*F2+C1*C4*F2*F2+F1*F2+F1*F2
150	
151	
152	
155	WALLE (0)107/
154	107 FUNDAL (//2/A/97 LL ROUIS///
155	DU LLU I-L94
156	110 WRIE (0,112) RUUR(1),RUUI(1)
157	112 FURMAL (1/A)C13+31UA/C13+31//
158	WKIIE (0,114) 16 ^K
159	114 FUKMAI (117,11)
160	WRIE (0,100)
161	106 FURMA((1H1,//)
162	125 DU 215 1=1+4

CARD

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CARI	
16	3 0A=F1
164	+ OB=F2*R00TR(I)+F3
16	j GC=F4*ROOTR(I)*FCOTR(I)+F5*ROOTR(I)+F6
16	5 0D=G1
16	/ OE=G2*ROOTR(I)+G3
16	3 OF=G4*RODTR(I)*RCGTR(I)+G5*RODTR(I)+G6
16	<pre>if (ABS(ROOTI(I)).LT.0.000001) ROOTI(I)=0.0</pre>
17) IF (RODTI(I)) 200,130,200
17	1 2CG WRITE (6,201)
17.	2 201 FORMAT (5X,18H ROOT IS IMAGINARY,/)
17	3 GO TO 215
17	4 130 SA=0B+40-0+0A+0C
17	j IF (SA) 200,140,140
17	5 140 ROOTA1=(-OB+SQRT(SA))/(2.0*GA)
17	7 ROOTA2=(-OB-SQRT(SA))/(2.0*OA)
17:	3 SB=0E+0E-4 •0 +0D +0F
17	9 IF (SB) 200,150,150
18	150 R00TB1=(-0E+SQRT(SB))/(2.0*0D)
18	ROOT B2= (-OE-SQRT(SB))/(2.0*OD)
18	2 WRITE (6,155) RCCTA1,ROOTA2,ROOTB1,ROCTB2
18	3 155 FORMAT (/,13X,8H ROOTA1=,F18.6,5X,8H ROOTA2=,F18.6,/,13X,8H ROOTB1
18	4 \$=,F18.6,5X,8H RC0TB2=,F18.6,/)
18	5 IF (ABS(ROOTA1-RCCTB1).LT.0.05) GO TO 160
18	5 IF (ABS(ROOTA1-ROOTB2)+LT+0+05) GO TO 160
18	7 IF (ABS(ROOTA2-RCOT B1).LT.0.05) GO TO 170
18	8 IF (ABS(ROOTA2-ROOTB2)+LT+0+05) GO TO 170
18	3 GD TO 215
19	0 160 WRITE (6.180) RCCTA1
19	1 180 FORMAT (2X.16H SECOND ROOT IS .F20.5./)
19	> ROOT=ROOT A1
19	3 GD TO 210
19	4 170 WRITE (6,180) RODIA2
19	
19	(-210 wr tr (-212))
19	7 212 FORMAT (14X-13H CENTER POINT-31X-13H CIRCLE POINT-/-8X-3H XA-19X-3
19	6 (H YA.19X.3H XB.19X.3H YB./)
10	
20	$\gamma = \gamma + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1 + 1 +$
. 20	y = (1) + (2) +
20	$Y_{B(1)} = 0 (A) + P(0) T_{B(1)} = 0 (A) + P(0) T_{B(1)}$
20	
20.	
20	
203	
20	
20	

APPENDIX C

SYNTHESIS PROGRAM FOR THREE FINITE POSITIONS

AND TWO FIRST INFINITESIMAL

DISPLACEMENTS

2	C ****	*****	****
- 2	C *		
3	C * .	SYNTHESIS OF SIX-LINK STEPHEN	ISON 2
4	C *	FUNCTION GENERATOR	
5	C *		
6	C *	THREE FINITE POSITIONS AND TWO VELOC	ITY RATIOS
7	C *		
8	ί¥ Γ#	INDET EINTTE DOTATIONS	TH 1 (T)
10	C *	INFOT FINITE RUTATIONS	[n1(1]
11	C *	DUTPLT FINITE ROTATIONS	PH1(I)
12	С *		
13	C *	ASSUMED ROTATIONS OF TERNARY	AL1(1)
14	C *		
15	C *	FIRST INFINITESTMAL DISPLACEMENT	DPHDTH(J)
17	C *	NUTPLE LINK LENGTH	or
18	C *		
19	C *	OUTPUT LINK INITIAL ANGLE	THE
20	С *		
21	C ****	*********	******
22	C		
23		UIMENSIUN XUUF(5),UUF(5),KUUIK(4),KUUII(4) DIMENSION VO(E) VO(E) VA(E) VA(E) VB(E) VO(E	
25		DIMENSION THI (5) PHi (5) $Aiii$ (5)	
26		DIMENSION H(16), H1(16), H2(16), R(4), P(4), Q(4)	
27		DIMENSION DXC(5), DYC(5), PXC(5), PYC(5)	
28		DIMENSION DPHDTH(5), DTHDAL(5)	
29	-	READ (5,5) J.K	
21	2	FURMA! (11911) WDITE (6.6) 1.K	
	6	RAIL (OFOS SYR	
22	Ŷ	FORMAL ITT. 15X. 34H VELOCITY RAILON ARE AT PO	STTICNS .TI SH AND .TI
32 33	\$	FORMAL (//,16X,34H VELUCITY RATIUS ARE AL PU ,//)	SITIONS ,11,5H AND ,11
32 33 34	\$	FORMAL (//,16%,34H VELUCILY RATIOS ARE AL PU ,//) READ (5,10) TH1(2),TH1(3)	SITIONS ,IL,5H AND ,IL
32 33 34 35	\$	FORMAI (77,163,34H VELUCIIY RATIUS ARE AT PU ,7/1) READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3)	SITIONS ,II,5H AND ,II
32 33 34 35 36	\$	FORMAI (77,163,34H VELUCIIY RATIUS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PH1(2),PH1(3) READ (5,10) AL1(2),AL1(3)	SITIONS ,IL,5H AND ,II
32 33 34 35 36 37	\$ 10	FORMAL (77,163,34H VELUCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) PEAD (5,20) OC.THE	SITIONS ,IL,5H AND ,IL
32 33 34 35 36 37 38	\$ 10	FORMAL (77,163,34H VELUCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PH1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) QC,THE READ (5,20) QC,THE	SITIONS ,IL,5H AND ,IL
32 33 34 35 37 38 37 38 37 38 37 38 39	\$ 10	FORMAL (77,153,34H VELUCIIY RATIOS ARE AT PU READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) QC,TFE READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K)	SITIONS ,IL,5H AND ,IL
32 33 34 35 36 7 38 9 0 1	\$ 10 20	FORMAL (77,153,34H VELUCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PL1(2),PH1(3) READ (5,10) ALI(2),ALI(3) FORMAT (2F10.0) READ (5,20) QC,TEE READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0)	SITIONS ,IL,5H AND ,II
32 334 35 37 38 37 37 37 37 37 37 37 37 37 37 37 37 37	\$ 10 20	FORMAL (77,163,34H VELUCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,10) ALI (2),ALI(3) FORMAT (2F10.0) READ (5,20) DC,TFE READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3)	SITIONS ,IL,5H AND ,II
32 334 35 36 7 39 9 0 1 23 34 5 37 39 9 0 1 23 34 5 37 39 9 0 1 23 34 5 37 39 5 37 5 37 5 37 5 37 5 5 5 7 5 7 5 7 5 7	\$ 10 20 11	FORMAT (77,163,34H VELUCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) QC,TFE READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X,	2F10.3,/)
32 334 356 378 390 142 344 44	\$ 10 20 11	FORMAT (77,163,34H VELOCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) QC,TFE READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) FORMAT (2F10.0) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X,	2F10.3,/)
32 33 33 33 33 33 33 33 33 33 33 33 33 3	\$ 10 20 11 12	FORMAT (77,15X,34H VELOCITY RATIOS ARE AT PU READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) DCHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,27H GUTPUT ROTATION ANGLES ARE,1X WPITE (6,13) N1(2).01(3)	2F10.3,/) ,2F10.3,/)
33333333334444444444	\$ 10 20 11 12	FORMAT (77,153,34H VELOCITY RATIOS ARE AT PU READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,27H GUTPUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3) FORMAT (9X,27H GUTPUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3)	2F10.3,/) ,2F10.3,/)
333333333344444444444444444444444444444	\$ 10 20 11 12 13	FORMAT (77,16X,34H VELOCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) AL1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,27H OUTPUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3) FORMAT (9X,23H TERNARY LINK ROTATIONS,5X,2F1 WRITE (6,14) QC,THE	2F10.3,/) ,2F10.3,/)
333333333344444444444444444444444444444	\$ 10 20 11 12 13 14	FORMAT (77,16X,34H VELOCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PL1(2),PH1(3) READ (5,10) ALI (2),ALI(3) FORMAT (2F10.0) READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,26H THOUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3) FORMAT (9X,23H TERNARY LINK ROTATIONS,5X,2F1 WRITE (6,14) QC,THE FORMAT (9X,13H CUTPUT LINK IS,F10.3,24H LONG	2F10.3,/) ,2F10.3,/) 0.3,//) AND AT AN ANGLE DF,F1
333333333444444444444444444444444444444	\$ 10 20 11 12 13 14 \$	FORMAT (77,16X,34H VELOCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PH1(2),PH1(3) READ (5,10) ALI (2),ALI(3) FORMAT (2F10.0) READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3) FORMAT (9X,23H TERNARY LINK ROTATIONS,5X,2F1 WRITE (6,14) QC,THE FORMAT (9X,15H CUTPUT LINK IS,F10.3,24H LONG 0.3,/)	SITIONS ,11,5H AND ,11 2F10.3,/) ,2F10.3,/) 0.3,//) AND AT AN ANGLE DF,F1
333333333444444444444444444444444444444	\$ 10 20 11 12 13 14 \$	FORMAT (77,15X,34H VELOCITY RATIOS ARE AT PU ,//) READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,20) QC,TFE READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,27H GUTPUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3) FORMAT (9X,23H TERNARY LINK ROTATIONS,5X,2F1 WRITE (6,15) DFFETH(J),DTHDAL(J) WRITE (6,15) DFFETH(J),DTHDAL(J)	2F10.3,/) ,2F10.3,/) 0.3,//) AND AT AN ANGLE DF,F1
333333333444444444444444444444444444444	\$ 10 20 11 12 13 14 \$ 15	FORMAT (77,16X,34H VELOCITY RATIOS ARE AT PU READ (5,10) TH1(2),TH1(3) READ (5,10) PF1(2),PH1(3) READ (5,10) AL1(2),AL1(3) FORMAT (2F10.0) READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DPHDTH(K),DTHDAL(K) FORMAT (2F10.0) WRITE (6,11) TH1(2),TH1(3) FORMAT (9X,26H INPUT ROTATION ANGLES ARE,2X, WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,27H GUTPUT ROTATION ANGLES ARE,1X WRITE (6,12) PH1(2),PH1(3) FORMAT (9X,27H GUTPUT ROTATION ANGLES ARE,1X WRITE (6,13) AL1(2),AL1(3) FORMAT (9X,35H TERNARY LINK ROTATIONS,5X,2F1 WRITE (6,15) DP+CTH(J),DTHDAL(J) FORMAT (9X,11H CPHDTH(J)=,F10.3,10X,11H DTHD WRITE (6,16) DD+CTH(J) THOAL(J)	SITIONS ,11,5H AND ,11 2F10.3,/) ,2F10.3,/) 0.3,//) AND AT AN ANGLE DF,F1 AL(J)=,F10.3,/)

	00000000111111111122222222233333333334444444444
	1234567890123456789012345678901234567890123456789012345678901234567890123456789012345678901
CARD	
55	XQ(1)=1.0
56	YQ(1) = 0.0
57	XM=0.0
58	YM=0.0
50	$TH_1(1) = 0$
20	
20	
01	
62	
63	
54	$(H_1(1) = ((H_1(1))^* 3 \cdot 1415 \cdot 9265) / 180 \cdot 0$
65	PH1(1)=(PH1(1)=3.14159265)/180.0
66	21 AL1(I)=(AL1(I)*3.14159265)/180.0
67	XC(1)=1.0+QC*COS(THE)
68	YC(1)=0.0+QC*SIN(THE)
69	DO 30 I=2,3
70	XQ(I)=1.0*CCS(-TH1(I))
71	YQ(I)=1.0*SIN(-TH1(I))
72	XC(I)=XC(1)*COS(PH1(I)-TH1(I))-YC(1)*SIN(PH1(I)-TH1(I))+XQ(I)-XQ(1
73	$s = c_0 S(PH1(1) - TH1(1)) + YQ(1) + SIN(PH1(1) - TH1(1))$
74	30 yr(1) = xc(1) + 5 in (PH1(1) - TH1(1)) + Yc(1) + CBS(PH1(1) - TH1(1)) + Yc(1) - xc(1)
75	(1 + (1) + (1) + (1) + (1)) + (0 + (1) + (0) + (0) + (1) + (0) +
76	DYC(1) = (YC(1)) = YO(1)) * (SIN(PH1(1)) = TH(1(1)) = SIN(PH1(1)) + TH(1)) * CPHDTH
70	
18	
19	
80	\$H(J))+(T((I)-TQ(I))+(SIN(PHI(J)-(II(J)-SIN(PHI(J)-)HI(J)+DPHD)H(
81	\$J])-XQ([)#CUS(-[H](J])+YQ([]#SIN(-[H](J))
82	DXC(K) = (XC(1) - XQ(1)) + (SIN(PH1(K) - IH1(K)) - SIN(PH1(K) - IH1(K)) + DPHDIH
83	\$ (K)) + (Y C (1) - Y Q (1)) * (CUS (PH1 (K) - TH1 (K)) - CUS (PH1 (K) - TH1 (K)) * OP HDTH (K
84	\$))+XQ(1)*SIN(-TH1(K))+YQ(1)*COS(-TH1(K))
85	DYC(K)=(XC(1)-XQ(1))*(-COS(PH1(K)-TH1(K))+COS(PH1(K)-TH1(K))*DPHDT
86	\$H(K))+(YC(1)-YQ(1))*(SIN(PH1(K)-TH1(K))-SIN(PH1(K)-TH1(K))*DPHDTH(
87	\$K)) - XQ(1) +COS(-TH1(K)) + YQ(1) * SIN(-TH1(K))
88	WRITE (6,17) DXC(J), DYC(J)
89	17 FBRMAT (9X,8H DXC(J)=,3X,F10.3,10X,8H DYC(J)=,3X,F10.3,/)
90	WRITE (6,18) DXC(K),DYC(K)
91	18 FORMAT (9X,8H DXC(K)=,3X,F10.3,10X,8H DYC(K)=,3X,F10.3//)
92	PXC(J) = DTHDAL(J) = XC(J)
93	PYC(J) = DT H CAL(J) * CYC(J)
94	PXC(K) = DTHEAL(K) * DXC(K)
05	PYC(K) = DT + DA + (K) + DYC(K)
94	
07	21 EODMAT (117.34 YO.187.34 YO.187.34 YO.187.34 YO.187.34
21	DO 22 T-1.2
98	
99	33 WRITE (0,32) AU(1), TU(1), TU(1), AU(1), TU(1)
100	32 FUKMAI (0X)FI2+ 895X)FI2+895X9FI2+895X9FI2+895X9FI2+89577
101	00 40 1=1,2
102	46 H(I)=XC(I+1)-XC(I)*COS(AL1(I+1))+YC(I)*SIN(AL1(I+1))
103	H(3)=PXC(J)+XC(1)*SIN(AL1(J))+YC(1)*CGS(AL1(J))
104	H(4)=PXC(K)+XC(1)*SIN(AL1(K))+YC(1)*COS(AL1(K))
105	DO 50 I=1,2
106	50 H(I+4)=YC(I+1)-YC(1)*COS(AL1(I+1))-XC(1)*SIN(AL1(I+1))
107	H(7)=PYC(J)-XC(1)*COS(AL1(J))+YC(1)*SIN(AL1(J))
108	H(8)=PYC(K)-XC(1)*COS(AL1(K))+YC(1)*SIN(AL1(K))

	000000001111111111222222222233333333334444444444
CARD	
109	D0 60 I=1-2
110	60 + (1+8) = - + (1) * (0 + (1+1)) - + (1+4) * SIN(AL1(1+1))
111	H(11) = -PXC(J) + CCS(A(1)) + PYC(J) + SIN(A(1)) + XC(J) + SIN(A(1)) + YC(J)
112	
112	H(1) = -DY(1, K) = (U(1) + DY(1, K) + DY(1, K) + Y(1,
11.4	
115	
114	70 1/14/2/=================================
117	
110	
110	UI12 1 = DAL (R F 7 2 2 1 1 1 1 1 1 2 1 1 = DAL (R) #CUC(V) #CU
120	ELECTION ALL AND ALL THE THE THE THE CONTRELIES ALL THE STALL THE STALL AND THE
120	
121	
122	
123	
124	
125	
120	
127	
128	\$) }+ (X((1) *X((3) *Y((1) *Y((1) ** C(3) - X((1) ** Y((3) + Y((1) ** X((3)) ** S(N(AL1(3))
129	
130	$R(4) = \{Y(1) + X(1) + X(1) + Y(1) +$
131	\$)] + {XC(1) * XC(K) + YC(1) * YC(K) - XC(1) * PYC(K) + YC(1) * PXC(K)] * SIN(AL1(K))
132	\$+XC(K)*PXC(K)+YC(K)*PYC(K)
133	DU 90 I=1,2
134	9C P(I)=1.0-CDS(ALI(I+1))
135	P(3)=SIN(AL1(J))
136	P(4)=SIN(AL1(K))
137	DO 100 I=1,2
138	100 Q(I)=SIN(AL1(I+1))
139	Q(3)=COS(AL1(J))
140	Q(4) = COS(AL1(K))
141	CALL SIMQ(H,R,4,KS)
142	CALL SIMQ(H1,P,4,KS1)
143	CALL SIMQ(H2,Q,4,KS2)
144	WRITE (6,99) KS
145	WRITE (6,99) KS1
146	WRITE (6,99) KS2
147	99 FORMAT (1X,11)
148	F1=Q(1)*Q(3)+Q(2)*Q(4)
149	F2=P(1)*Q(3)+Q(1)*P(3)+P(2)*Q(4)+Q(2)*P(4)
150	F3=R(1)*Q(3)+G(1)*R(3)+Q(2)*R(4)+Q(4)*R(2)
151	F4=P(1)*P(3)+P(2)*P(4)
152	F5=P(1)*R(3)+P(3)*R(1)+P(2)*R(4)+P(4)*R(2)-1.C
153	F6=R(1)*R(3)+R(2)*R(4)
154	G1=Q(1)*Q(4)-Q(2)*Q(3)
155	$G_{2}=P(1)*Q(4)+C(1)*P(4)-P(2)*Q(3)-Q(2)*P(3)$
156	G3=R(1)*Q(4)+Q(1)*R(4)-R(2)*Q(3)-Q(2)*R(3)-1.0
157	$G_{4=}P(1)*P(4)-P(2)*P(3)$
158	G5=P(1)*R(4)+R(1)*P(4)-P(2)*R(3)-P(3)*R(2)
159	$G_{6=R(1)*R(4)-R(2)*R(3)}$
160	XCDF(1)=F1*F1*G6*G6+G3*G3*F1*F6-G3*G6*F1*F3+G1*G4*F3*F3+G1*G1*F6*F
161	\$ 6-G 1*G 3*F 3*F 6-2. 0*G 1*G 5*F 1*F 6
162	XCDF(2)=2.0*G5*G6*F1*F1+2.0*G2*G3*F1*F6+G3*G3*F1*F5-G3*G6*F1*F2-G2

CARD \$*G6*F1*F3-G3*G5*F1*F3+2.0*G1*G6*F2*F3+G1*G5*F3*F3+2.0*G1*G1*F5*F6-163 \$G1 * G3 * F2 * F6 - G1 * G2 * F3 * F6 - G1 * G3 * F3 * F5 - 2,0 * G1 * G5 * F1 * F6 - 2,0 * G1 * G6 * F1 * F 164 165 \$5 XCDF(3)=2.C*G4*G6*F1+F1+G5*G5*F1+F1+G2*G2*F1*F6+2.0*G2*G3+F1*F5+G3 166 \$+G3 + F1 + F4 - G2 + G6 + F1 + F2 - G3 + G5 + F1 + F2 - G2 + G5 + F1 + F3 - G3 + G4 + F1 + F3 + G1 + G6 + F2 167 \$*F2+2.0*G1*G5*F2*F3+G1*G4*F3*F3+2.0*G1*G1*F4*F6+G1*G1*F5*F5-G1*G2* 168 \$F2*F6-G1*G3*F2*F5-G1*G2*F3*F5-G1*G3*F3*F4-2.0*G1*G6*F1*F4-2.0+G1*G 169 \$5*F1*F5-2.0*G1*G4*F1*F6 170 XCOF(4)=2.0*G4*G5*F1*F1+G2*G2*F1*F5+2.0*G2*G3*F1*F4-G2*G5*F1*F2+G3 171 \$*G4*F1*F2-G2*G4*F1*F3+G1*G5*F2*F2+2.0*G1*G4*F2*F3+2.0*G1*G1*F4*F5-172 173 \$G1*G2*F2*F5-G1*G3*F2*F4-G1*G2*F3*F4-2.0*G1*G5*F1*F4-2.0*G1*G4*F1*F 174 \$5 175 XCDF(5)=G4*G4*F1*F1+G2*G2*F1*F4-G2*G4*F1*F2+G1*G4*F2*F2+G1*G1*F4*F 176 \$4-G1*G2*F2*F4-2.0*G1*G4*F1*F4 177 CALL POLRT(XCOF,COF,4,ROOTR,ROOTI,IER) 178 WRITE (6,109) 179 109 FORMAT (/,27X,9H L1 RCOTS,/) DD 110 I=1,4 180 110 WRITE (6,112) RECTR(I), ROOTI(I) 181 112 FORMAT (17X,E13.5,10X,E13.5,/) 182 WRITE (6,114) IER -183 184 114 FORMAT (1X.II) WRITE (6,106) 185 106 FORMAT (1H1,//) 186 187 125 DO 215 I=1,4 188 0 A= F 1 OB=F2*ROOTR(I)+F3 189 OC = F4 * ROOTR(I) * RCOTR(I) + F5 * ROOTR(I) + F6190 191 0.0 = G1DE=G2 * RODTR (I) + G3192 OF=G4*ROOTR(I)*ROOTR(I)+G5*ROOTR(I)+G6193 IF (ABS(RODTI(I)).LT.0.000001) ROOTI(I)=0.0 194 195 IF (ROOTI(I)) 200,130,200 196 200 WRITE (6,201) 201 FORMAT (5X, 18H ROOT IS IMAGINARY,/) 197 GO TO 215 198 199 130 SA=08*08-4.0*0A*0C 200 IF (SA) 200,140,140 201 140 R00 TA1=(-08+SQRT(SA))/(2.0*0A) 202 ROOTA2=(-OB-SORT(SA))/(2.0*OA)203 SB=0E*0E-4.0*0D*CF 204 IF (SB) 200,150,150 205 150 ROOTB1= (-OE+SQRT(SB))/(2.0+00) ROOTB2=(-OE-SQRT(SB))/(2.0*OD) 206 WRITE (6,155) ROOTA1, ROOTA2, ROOTB1, ROOTB2 207 155 FORMAT (/,13X,8+ ROOTA1=,F18.6,5X,8H ROOTA2=,F18.6,/,13X,8H ROOTB1 208 \$=,F18.6.5X,8H RECTB2=,F18.6,/) 209 IF (ABS(ROOTA1-ROOTB1).LT.0.05) GO TO 160 210 IF (ABS(RDOTA1-ROOT B2).LT.0.05) GO TO 160 211 IF (ABS(ROOTA2-ROOTB1).LT.0.05) GD TO 170 212 IF (ABS (ROOTA2-ROOTB2).LT.0.05) GO TO 170 213 214 GO TO 215 160 WRITE (6,180) ROOTA1 215 180 FORMAT (2X,16H SECOND ROOT IS ,F20.5,/) 216

	000000000111111111112222222223333333333
CARD	
217	ROOT = ROOT A 1
218	GO TO 210
219	170 WRITE (6,180) ROOTA2
220	ROOT = ROOT A2
221	210 WRITE (6,212)
222	212 FORMAT (14X,13H CENTER POINT,31X,13H CIRCLE PCINT,/,8X,3H XA,19X,3
223	\$H YA,19X,3H XB,19X,3H YB,/}
224	XA(I)=R(1)+P(1)+RCOTR(I)+Q(1)*ROOT
225	YA(I)=R(2)+P(2)*R00TR(I)+Q(2)*R00T
226	XB(I)=R(3)+P(3)+ROTR(I)+Q(3)+ROOT
227	YB(I)=R(4)+P(4)*ROOTR(I)+Q(4)*ROOT
228	WRITE (6,230) XA(I),YA(I),XB(I),YB(I)
229	230 FORMAT (4F20.8,/,33X,20H XXXXXXXXXXXXXXXXX,//)
230	215 CONTINUE
231	STOP
232	END

.

APPENDIX D

SYNTHESIS PROGRAM FOR THREE FINITE POSITIONS

AND ONE SECOND INFINITESIMAL

DISPLACEMENT

C 40 D	C000000001111111111222222222233333333333	7777778 34567890
1	C ************************************	*****
2 3 4	C * SYNTHESIS OF SIX-LINK STEPHENSON 2 C * FUNCTION GENERATOR	* *
567	C * C * THREE FINITE POSITIONS AND ONE ACCELERATION RATIO C *	*
8 9	Č * C * INPUT FINITE ROTATIONS THI(I) C *	* *
11	C * OUTPUT FINITE ROTATIONS PH1(I)	*
12	C * C * ASSUMED ROTATIONS OF TERNARY AL1(I) C *	* *
15	C * SECOND INFINITESIMAL DISPLACEMENT DDPHTH(J)	*
16	C * DUTPUT LINK LENGTH QC	*
18 19	C * OUTPUT LINK INITIAL ANGLE THE	*
20 21	C ************************************	*****
21 22 22 24 26 7 89 31 23 33 34 56 7 89 01 23 33 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 24 56 7 89 01 24 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 01 23 34 56 7 89 00 12 34 56 56 7 89 00 12 34 56 56 7 89 00 12 34 56 56 89 00 12 34 56 56 7 89 00 12 34 56 56 33 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 34 55 56 7 89 00 12 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	C ************************************	******
46 47	WRITE (6,13) ALI(2),ALI(3) 13 FORMAT (9X,23H TERNARY LINK ROTATIONS,5X,2F10.3,//)	
48 49	WRITE (6,14) CC,THE 14 FORMAT (6X,15H CLTPUT LINK IS,F10.3,24H LONG AND AT AN ANGLE OF,F1	
50 51 52	50.3,/) WRITE (6,15) DP+CTH(J),DTHDAL(J) 15 FORMAT (9X,11H DPHDTH(J)=,F10.3,10X,11H DTHDAL(J)=,F10.3,/)	
53 54	<pre>write (6,16) DDPhth(J],DDTHAL(J) 16 FORMAT (9X,11+ CDPHth(J)=,F10.3,10X,11+ DDTHAL(J)=,F10.3,//)</pre>	
	0000000011111111112222222223333333334444444444	
----------------	--	
CARD		
55	XQ(1)=1.0	
56	YQ(1)=0.0	
57	X M=0 •C	
58	YM = 0 - 0	
50	$H_{1}(1)=0.0$	
60		
41		
	TUE-17-000 TUE-17-E000 141502451/100 0	
62		
60		
04		
65	$PH(1) = (PH(1)^{+5} \cdot 14153205) / 100 \cdot 0$	
66	21 ALI(1)=(ALI(1)*3.1413920377100.0	
61		
68	T(1)=0.0+QC#SIN(1HE)	
69		
70	$XQ(1) = 1.0 \times US(-1) + 1(1)$	
$\frac{n}{2}$		
72	XC(1) = XC(1) + CUS(PHI(1) - (HI(1)) + C(1) + SIN(PHI(1) - (HI(1)) + XQ(1) - XQ(1))	
(3	$s_{1} = c_{1} + c_{1$	
74	30 YC(1)=XC(1)=X N(PH1(1)-1H1(1))+YC(1)=C(1)=C(1)+C(1)+YC(1)-XC(1)	
75	$s_{j} = s_{i} (PH(1) - HI(1)) - YQ(1) = COS(PH(1) - HI(1))$	
76	DXC(J) = (XC(T) - XQ(T)) * (SIN(PHI(J) - HI(J)) - SIN(PHI(J) - HI(J)) * DPHD H	
77	\$(J))+(Y(1)-Y(1))*(CCS(PH1(J)-TH1(J))+COS(PH1(J)+TH1(J))*DPHDTH(J	
78	\$))+XQ(1)*SIN(-TH1(J))+YQ(1)*CUS(-TH1(J))	
79	DYC(J) = (XC(I) - XQ(I)) * (-COS(PHI(J) - THI(J)) + COS(PHI(J) - THI(J)) * DPHDT	
80	H(J) + (YC(I) - YQ(I)) + (SIN(PHI(J) - 1HI(J)) - SIN(PHI(J)) + DPHDTH(J)) + DPHDTH(J)) + DPHDTH(J) + DPHDTH(J)) + DPHDTH(J) + DPHDTH(J)) + DPHDTH(J) + DPHDTH(J)) + DPHDTH(J)) + DPHDTH(J) + DPHDTH(J)) + DPHDTH(D)) + DPHDTH(D)) + DPHDTH(D)	
81	\$J })-XQ(1)*CUS(-TH1(J))+YQ(1)*SIN(-TH1(J))	
82	$DDXC(J) = \{XC(I) - XC(I)\} * \{-CUS(PHI(J) - IHI(J)\} + 2.0 * CUS(PHI(J) - IHI(J)) * (-CUS(PHI(J) - IHI(J)) + 2.0 * CUS(PHI(J) - IHI(J)) \}$	
83	\$DPHDTH(J)-COS(PH1(J)+TH1(J))*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-TH1(J)	
84	\$)*DDPHTH(J))+(YC(1)-YQ(1))*(SIN(PH1(J)-1H1(J))-2.0*SIN(PH1(J)-TH1(
85	\$J})*DPHDTH(J)+SIN(PH1(J)-TH1(J))*DP+DTH(J)*COS(PH1(J)-TH	
86	\$1(J))*DDPHTH(J))-XQ(1)*COS(-TH1(J))+YQ(1)*SIN(-TH1(J))	
87	DDYC(J) = (XC(1) - XQ(1)) + (-SIN(PH1(J) - TH1(J)) + 2.0 + SIN(PH1(J) - TH1(J)) +	
88	\$DPHDTH(J) - SIN(PH1(J) - TH1(J)) *DPHDTH(J) *DPHDTH(J) + COS(PH1(J) - TH1(J)	
89	\$)*DDPHTH(J))+(YC(1)-YQ(1))*(-CUS(PH1(J)-TH1(J))+2.0*COS(PH1(J)-TH1	
90	\$(J))*DPHDTH(J)-CCS(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-T	
91	\$H1(J)}*D2PHTH(J)}-XQ(1)*SIN(-TH1(J))-YQ(1)*COS(-TH1(J))	
92	PXC(J)=DTHDAL(J)*DXC(J)	
93	PYC(J) = DT HDAL(J) * DY C(J)	
94	PPXC(J)=DTHDAL(J)+DTHDAL(J)+DDXC(J)+DDTHAL(J)+DXC(J)	
95	PPYC(J)=DTHCAL(J)*DTHDAL(J)*DDYC(J)+DDTHAL(J)*DYC(J)	
96	WRITE (6,17) DXC(J), DYC(J)	
97	17 FORMAT (9X,8H DXC(J)=,3X,F10.3,10X,8H DYC(J)=,3X,F10.3,/)	
9 8	WRITE (6,18) DDXC(J),DDYC(J)	
99	18 FORMAT (9X,9H DCXC(J)=,2X,F10.3,10X,9H DDYC(J)=,2X,F10.3,//)	
100	WRITE (6,31)	
101	31 FORMAT (11X,3H XÇ,18X,3H YQ,18X,3H XC,18X,3H YC,/)	
102	DO 33 I=1,3	
103	33 WRITE (6,32) XQ(1),YQ(1),XC(1),YC(1)	
104	32 FORMAT (6X,F12.8,9X,F12.8,9X,F12.8,9X,F12.8,/)	
105	D0 40 I=1,2	
106	40 H(I)=XC(I+1)-XC(1)*COS(AL1(I+1))+YC(1)*SIN(AL1(I+1))	
107	H(3)=PXC(J)+XC(1)*SIN(AL1(J))+YC(1)*COS(AL1(J))	
108	H(4)=PPXC(J)+XC(1)*COS(AL1(J))-YC(1)*SIN(AL1(J))	

	0000000011111111112222222223333333334444444445555555555
CARD	
109	D0 50 I=1,2
110	50 H(I+4)=YC(I+1)-YC(1)*COS(AL1(I+1))-XC(1)*SIN(AL1(I+1))
111	H(7)=PYC(J)-XC(1)*COS(AL1(J))+YC(1)*SIN(AL1(J))
112	H(8)=PPYC(J)+XC(1)*SIN(AL1(J))+YC(1)*COS(AL1(J))
- 113	DO 60 I=1,2
114	6C H(I+8)=-H(I)*COS(AL1(I+1))-H(I+4)*SIN(AL1(I+1))
115	H(11)=-PXC(J)*COS(AL1(J))-PYC(J)*SIN(AL1(J))+XC(J)*SIN(AL1(J))-YC(
116	\$J}*COS(AL1(J})
117	H(12)=-PPXC(J)*COS(AL1(J))-PPYC(J)*SIN(AL1(J))+2.0*PXC(J)*SIN(AL1(
118	\$J])-2.0*PYC(J)*CGS(AL1(J))+XC(J)*CGS(AL1(J))+YC(J)*SIN(AL1(J))
119	DO 70 I=1,2
120	70 H(I+12)=-H(I+4)*COS(AL1(I+1))+H(I)*SIN(AL1(I+1))
121	H(15)=PXC(J)*SIN(AL1(J))-PYC(J)*COS(AL1(J))+XC(J)*COS(AL1(J))+YC(J
122	\$)*SIN(AL1(J))
123	H(16)=PPXC(J)*SIN(ALI(J))-PPYC(J)*CUS(ALI(J))+2.0*PXC(J)*CUS(AL1(J)
124	\$)] + 2. 0* PYC(J) * SIN(ALI(J) / - XC(J) * SIN(ALI(J)) + YC(J) * CUS(ALI(J))
125	
126	72 H1(1) = H(1)
127	JO 74 1=1,16
128	74 H2(1)=H(1)
129	
130	80 K(1)=0.54(H(1)+H(1)+H(1+4)+H(1+4)+
120	
132	\$) }+ (AU(1) * AU(3) * AU(1) * AU(3) * AU(1) * AU(3) *
133	
134	$U_{\text{DI}} = 2 \cdot 0 + 1 \cdot (1) + 7 \cdot (1)$
132	\$AU11#AU31#TU11#TU30 DD2=0 A4VC111#D1VC11A=0 A4VC111#DVC111#DVC111#DVC111#DVC111#D0VC111_
127	
120	\$76(1)##C0(1)#10(1)#10(0) \$76(-CO(1)1/(1)#C011#1(1)#C012#YC/(1)#C0YC/(1)#C0YC/(1)#C0YC/(1)#C0YC/(1)#C0YC/(1)#C0YC/(1)#C0YC/(1)#C0YC/(1)#C
120	<pre>k(f)=003(#L1(3)**DVC(1)**DVC(0)************************************</pre>
140	
141	90 p(1) = 1, 0 = 0.05 (A) 1(1+1)
142	
143	P(4) = COS(4) I(4)
144	
145	100 o(1) = STN(A 1(1+1))
146	Q(3) = COS(ALI(J))
147	Q(4) = -STN(AL1(4))
148	CALL SIMO(H - R - 4 - KS)
149	CALL SIMO(H1+P+4+KS1)
150	CALL SIM0(H2+0+4+K52)
151	WRITE (6.99) KS
152	WRITE (6,59) KS1
153	WRITE (6,99) KS2
154	99 FORMAT (1X,11)
155	F 1=Q(1) +Q(2) +Q(2) +Q(4)
156	F2=P(1)*Q(3)+Q(1)*P(3)+P(2)*Q(4)+Q(2)*P(4)
157	F3=R(1)*Q(3)+Q(1)*R(3)+Q(2)*R(4)+Q(4)*R(2)
158	F4=P(1)*P(3)+P(2)*P(4)
159	F5=P(1)*R(3)+P(3)*R(1)+P(2)*R(4)+P(4)*R(2)~1.0
160	F6=R(1)*R(3)+R(2)*R(4)
161	G1=Q(1)*Q(4)-Q(2)*Q(3)
162	G2=P(1)*Q(4)+Q(1)*P(4)-P(2)*Q(3)-Q(2)*P(3)

163	$G_3=R(1)*Q(4)+Q(1)*R(4)-R(2)*Q(3)-Q(2)*R(3)-1.0$
164	G4=P(1)*P(4)-P(2)*P(3)
165	G5=P(1)*R(4)+R(1)*P(4)-P(2)*R(3)-P(3)*R(2)
166	G6=R(1)*R(4)-R(2)*R(3)
167	XCOF(1)=F1*F1*G6*G6+G3*G3*F1*F6+G3*G6*F1*F3+G1*G6*F3*F3+G1*F6*F
168	\$6-G1*G3*F3*F6-2.0*G1*G6*F1*F6
169	XCOF(2)=2:0*G5*G6*F1*F1+2:0*G2*G3*F1*F6+G3*G3*F1*F5-G3*G6*F1*F2-G2
170	\$*G6*F1*F3-G3*G5*F1*F3+2.0*G1*G6*F2*F3+G1*G5*F3+F3+2.0*G1*G1*F5+F6-
171	\$G1*G3*F2*F6-G1*G2*F3*F6-G1*G3*F3*F5-2.0*G1*G5*F1*F6-2.0*G1*G6*F1*F
172	\$5
173	XC0F(3)=2.0*G4*G6*F1*F1+G5*G5*F1*F1+G2*G2*F1*F6+2.0*G2*G3*F1*F5+G3
174	\$*G3*F1*F4-G2*G6*F1*F2-G3*G5*F1*F2-G2*G5*F1*F3-G3*G4*F1*F3+G1*G6*F2
175	\$*F2+2.0*G1*G5*F2*F3+G1*G4*F3*F3+2.0*G1*G1*F4*F6+G1*G1*F5*F5-G1*G2*
176	\$F2*F6-G1*G3*F2*F5-G1*G2*F3*F5-G1*G3*F3*F4-2.0*G1*G6*F1*F4-2.0*G1*G
177	\$5*F1*F5-2.0*G1*G4*F1*F6
178	XCD F(4) = 2,0*G4*G5*F1*F1+G2*G2*F1*F5+2,0*G2*G3*F1*F4-G2*G5*F1*F2-G3
179	\$* G4 *F1 *F2 -G2 *G4 *F1 *F3+G1 *G5 *F2 *F2 +2 .0 *G1 *G4 *F2 *F3 +2 .0 *G1 *G1 *F4 *F5 +
180	\$61+62+E2+E5-61+63+E2+E4-61+62+E3+E4+2=0+61+65+E1+E4-2=0+61+64+E1+E
181	
102	
102	$\mathbf{x}_{0} = (\mathbf{y}_{1}) - \mathbf{y}_{1} = (\mathbf{x}_{1}) + \mathbf{y}_{2} = (\mathbf{x}_{1}) + \mathbf{y}$
100	
104	
104	MRITE (01107) 100 EODMAT (/ 27Y OH 13 DÉGTE./)
100	DO 10 7-0 4
187	
188	$\frac{110}{110} \text{ while } (0,112) \text{ Ruch (1), Ruch (1)}$
189	112 FURMAL (17/A)E13-3310/A)E13-33777
190	WRITE ($0,114$) ick
191	114 FURMA((1X)11)
192	WRITE (6,106)
193	106 FORMAT (1H1,//)
194	125 00 215 1=1.4
195	
196	0B=F2*R00TR(1)+F3
197	OC=F4*RODTR(I)*RODTR(I)+F5*RODTR(I)+F6
198	0D=G1
199	OE=G2*ROOTR(I)+G3
200	DF=G4*R03TR(I)*R00TR(I)+G5*R00TR(I)+G6
201	IF (ABS(ROOTI(I)).LT.0.COOOOO1) ROOTI(I)=0.0
202	IF (ROOTI(I)) 20C,130,200
203	200 WRITE (6,201)
204	201 FORMAT (5X,18H ROOT IS IMAGINARY,/)
205	GO TO 215
206	130 SA=0B+0B-4.0*0A*0C
207	IF (SA) 200,140,140
208	140 R00TA1=(-08+SQRT(SA))/(2.0*0A)
209	ROOTA2 = (-OB-SQRT(SA))/(2.0*OA)
210	SB=0E+0E-4.0+0D+0F
211	IF (SB) 200,150,150
212	15C R00TB1=(-0E+SQRT(SB))/(2+0*0D)
213	ROOTB2 = (-OE - SORT(SB))/(2.0*OD)
214	WRITE (6,155) ROOTA 1, ROOTA 2, ROOTB 1, ROOTB 2
215	155 FORMAT (/.13X.8+ ROOTAL=,F18.6,5X.8H ROOTA2=,F18.6,/.13X.8H ROOTAL
216	<pre>s=.F18.6.5X.8H.RGTB2=.F18.6./)</pre>
210	4. 1. TOPOTONY CO. ICALDE - 1. TOPOTIC

G3=R(1)*Q(4)+Q(1)*R(4)-R(2)*Q(3)-Q(2)*R(3)-1.0

CARD

	0000000011111111112222222222222333333333
	123456789012345867890123458890123458890123458890123458898012889888888888888888888888888888888
CARD	
217	IE (ABS(R00TA1-R00TB1).(T.0.05) G0 T0 160
218	IF (ABS(ROUTA1-ROUTB2).LT.0.05) GO TO 160
219	IF (ABS(ROOTA2-ROOTB1).LT.0.05) GO TO 170
220	IF (ABS(RODTA2-ROOTB2).LT.0.05) GO TO 170
221	GO TO 215
222	16C WRITE (6,180) RCCTA1
223	180 FORMAT (2X,16H SECOND ROOT IS ,F20.5,/)
224	ROOT=ROOT A1
225	GO TO 210
226	170 WRITE (6,180) RCOTA2
227	RCOT=ROOTA2
228	210 WRITE (6,212)
229	212 FORMAT (14X,13H CENTER POINT,31X,13H CIRCLE POINT,/,8X,3H XA,19X,3
230	\$H YA,19X,3H XB,19X,3H YB,/)
231	XA(I)=R(1)+P(1)*ROD TR(I)+Q(1)*RODT
232	YA(I)=R(2)+P(2)*RODTR(I)+Q(2)*RODT
233	XB(I)=R(3)+P(3)*RODTR(I)+Q(3)*ROOT
234	YB(I)=R(4)+P(4)*ROOTR(I)+Q(4)*ROOT
235	WRITE (6,230) XA(I),YA(I),XB(I),YB(I)
236	230 FORMAT (4F20.8,/,33X,20H XXXXXXXXXXXXXXXXXX,//)
237	215 CONTINUE
238	STOP

239 END

APPENDIX E

SYNTHESIS PROGRAM FOR TWO FINITE POSITIONS,

ONE FIRST INFINITESIMAL DISPLACEMENT

AND ONE SECOND INFINITESIMAL

DISPLACEMENT

7

.

1	00000000111111111222222233333333334444444444	55555555555666666666666666666666666666	677777777778 901234567890	
1 (*******	******	* * * * * * * * * * * *	
2 (3 (4 (* SYNTHESIS OF SIX-LINK STEPHENS * FUNCTION GENERATOR	50N 2	* * * *	
5 (6 (7 (8 (* TWO FINITE POSITIONS, * ONE VELOCITY RATID, AND CNE ACCELERA *	TION RATIO	* * *	
9 (* INPUT FINITE ROTATION	TH1(1)	*	
11 (12 (* OUTPUT FINITE ROTATION	PH1(I)	*	
13 (* ASSUMED ROTATION OF TERNARY	AL1(I)	* *	
15 (* FIRST INFINITESIMAL DISPLACEMENT	DP HD TH (K)	* *	
18 0	* SECOND INFINITESIMAL DISPLACEMENT	DDPHTH (J)	*	
20 0	* OUTPLT LINK LENGTH	QC	*	
22 0	* OUTPUT LINK INITIAL ANGLE	THE	*	
24 (* * *****	****	*	
26 0	DIMENSION XCOF(5).COF(5).RDOTR(4).ROOTI(4)			
28	DIMENSION XQ(5),YQ(5),XA(5),YA(5),XB(5),YB(5)	5),XC(5),YC(5)		
30 31	DIMENSION H(16), H1(16), H2(16), R(4), P(4), Q(4) DIMENSION DPHDTH(5), DCPHTH(5), DTHDAL(5), DCT) HAL (5)		
32 33	DIMENSION DXC(5),DYC(5),DDXC(5),DDYC(5) DIMENSION PXC(5),PYC(5),PPXC(5),PPYC(5)			•
34 35	READ (5,5) J,K 5 Format (11,11)			
36 37	WRITE (6,6) K,J 6 FORMAT (//,5X,32H VELOCITY RATIO IS AT POSI	ITION , 11,39H AND AG	CEL	
38 39	<pre>\$ERATION RATIO IS AT POSITION ,I1,//) READ (5,10) TH1(2)</pre>			
40 41	READ (5,10) PH1(2) READ (5,10) AL1(2)			
42 43	10 FORMAT (F10.0) READ (5.20) QC.THE			
44 45	READ (5,20) DPHDTH(J),DTHDAL(J) READ (5,20) DDPHTH(J),DDTHAL(J)			
46 47	READ (5,20) DPHDTH(K),DTHDAL(K) 20 FORMAT (2F10.0)			
48 49	WRITE (6,11) TH1(2) 11 FORMAT (9X.24H INPUT RUTATION ANGLE IS,2X.FI	10.3,/)		
50 51	WRITE (6,12) PH1(2) 12 FORMAT (9X,25H OUTPUT ROTATION ANGLE IS,1X,F	=10.3,/)		
52	WRITE (6,13) AL1(2)	.3,/)		
-				

123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890 CARD 55 14 FORMAT (6X, 15H OLTPUT LINK IS, F10.3, 24H LONG AND AT AN ANGLE OF, F1 56 \$0..3 1/1 57 WRITE (6,15) DP+DTH(J),DTHDAL(J) 58 15 FORMAT (9X,11H DPHD TH(J)=,F10.3,10X,11H DTHDAL(J)=,F10.3,/) WRITE (6.16) DOPETH(J). DDTHAL(J) 59 16 FORMAT (9X,11H DDPHTH(J)=,F10.3,10X,11H DDTHAL(J)=,F10.3,/) 60 61 WRITE (6,19) DPHDTH(K), DTHDAL(K) 19 FORMAT (9X,11h CPHOTH(K)=, F10.3,10X,11H DTHDAL(K)=, F10.3,//) 62 63 XQ(1)=1.0YQ(1)=0.0 64 65 XM=0.0 YM= 0.0 66 67 TH1(1)=0.0 PH1(1)=0.0 68 AL1(1)=0.0 69 70 THE=(THE*3.14159265)/180.0 71 1=2 TH1(I)=(TH1(I)*3.14159265)/180.0 72 73 PH1 (I) = (PH1(I) *3.14159265)/180.0 74 AL1(I)=(AL1(I)*3.14159265)/180.0 75 XC(1)=1.0+QC*CDS(THE) 76 $YC(1) = 0.0 + QC \times SIN(THE)$ 77 XQ(I) = 1.0 * COS(-TH1(I))78 YQ(I)=1.0*SIN(-TH1(I)) 79 XC(I)=XC(1)*COS(PH1(I)-TH1(I))-YC(I)*SIN(PH1(I)-TH1(I))+XQ(I)-XQ(I) 80 \$)*COS(PH1(I)-TH1(I))+YQ(1)*SIN(PH1(I)-TH1(I)) YC(I)=XC(1)*SIN(PE1(I)-TH1(I))+YC(1)*COS(PH1(I)-TH1(I))+YQ(I)-XQ(1 81 \$)*SIN(PH1(I)-TH1(I))-YQ(1)*COS(PH1(I)-TH1(I)) 82 83 DXC(J)=(XC(1)-XQ(1))*(SIN(PH1(J)-TH1(J))-SIN(PH1(J)-TH1(J))*DPHDTH\$ (J)) + (YC(1) - YQ(1)) * (COS (PH1 (J) - TH1 (J)) - COS (PH1 (J) - TH1 (J)) * DPHDTH (J 84 \$})+XQ(1)*SIN(-TH1(J))+YQ(1)*COS(-TH1(J)) 85 DYC(J) = (XC(1) - XQ(1)) * (-COS(PH1(J) - TH1(J)) + COS(PH1(J) - TH1(J)) * DPHDT86 87 \$H(J))+(YC(1)-YQ(1))*(SIN(PH1(J)-TH1(J))-SIN(PH1(J)-TH1(J))*DPHDTH(88 \$J))-XQ(1)*COS(-TH1(J))+YQ(1)*SIN(-TH1(J)) $DDx C(J) = (xC(1) - xQ(1)) * (-COS(PH1(J) - TH1(J)) + 2 \cdot 0 * COS(PH1(J) - TH1(J)) *$ 89 \$DPHDTH(J)-COS(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-TH1(J) 90 \$)*DDP HT H(J))+(YC(1)-YQ(1))*(SIN (PH1(J) - TH1(J))-2.0*SIN (PH1(J) -TH1(91 \$J}) *DPHDTH(J)+SIN(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-COS(PH1(J)-TH 92 \$1(J))*DDPHTH(J))-XQ(1)*COS(-TH1(J))+YQ(1)*SIN(-TH1(J)) 93 $DDYC(J) = {XC(1) - XQ(1)} * (-SIN(PH1(J) - TH1(J)) + 2.0*SIN(PH1(J) - TH1(J)) *$ 94 95 \$DPHDTH(J) -SIN(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)+COS(PH1(J)-TH1(J) 96 \$) *DDP HTH(J))+(YC(1)-YQ(1))*(-C0S(PH1(J)-TH1(J))+2.0*COS(PH1(J)-TH1 97 \$(J))*DPHDTH(J)~COS(PH1(J)~TH1(J))*DPHDTH(J)*DPHDTH(J)~SIN(PH1(J)~T 98 \$H1(J))*D0PHTH(J))-XQ(1)*SIN(-TH1(J))-YQ(1)*COS(-TH1(J)) 99 DXC(K) = (XC(1) - XO(1)) * (SIN(PH1(K) - TH1(K)) - SIN(PH1(K) - TH1(K)) * DPHDTH\$(K))+(YC(1)-YQ(1))*(COS(PH1(K)-TH1(K))+COS(PH1(K)-TH1(K))*DPHDTH(K 100 101 \$))+XQ(1)#SIN(-TH1(K))+YQ(1)*COS(-TH1(K)) 102 DYC(K)=(XC(1)-XQ(1))*(-COS(PH1(K)-TH1(K))+COS(PH1(K)-TH1(K))*DPHDT \$H(K)}+(YC(1)-YQ(1))*(SIN(PH1(K)-TH1(K))-SIN(PH1(K)-TH1(K))*DPHDTH(103 \$K)) - XQ(1) *COS(-TH1(K)) + YQ(1) * SIN(-TH1(K)) 104 WRITE (6,17) DXC(J), DYC(J) 105 17 FORMAT (9x,8H DXC(J)=,3X,F10.3,10X,8H DYC(J)=,3X,F10.3,/) 106 107 WRITE (6,21) DD XC(J),DD YC(J) 108

21 FORMAT (9X,9H DCXC(J)=,2X,F10.3,10X,9H DDYC(J)=,2X,F10.3./)

N

	123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890
CARD	
109	WRITE (6,18) CXC(K),DYC(K)
110	18 FORMAT (9X,8H DXC(K)=,3X,F10.3,10X,8H DYC(K)=,3X,F10.3,///)
111	PXC(J) = DTHDAL(J) + DXC(J)
112	PYC(J)=DTHCAL(J)=DYC(J)
113	PPXC(J)=DDTHAL(J)+DXC(J)+DTHDAL(J)+DTHDAL(J)+DDXC(J)
114	PPYC(J)=DDTHAL(J)*DYC(J)+DTHDAL(J)*DTHDAL(J)*DDYC(J)
115	PXC(K) = DT H CAL(K) + CXC(K)
116	PYC(K) = DTHDAL(K) + DYC(K)
117	
118	31 FURMAL (11X,3H XQ,18X,3H YQ,18X,3H XC,18X,3H YC,7)
119	
120	33 WRITE (6,32) XQ(1), TQ(1), XC(1), TC(1)
121	32 FURMA1 (0X,F12.8,9X,F12.8,9X,F12.8,9X,F12.8,77)
122	
123	$H(1) = X_{1}(1+1) - X_{2}(1) + CUS(ALI(1+1)) + TC(1) + SIN(ALI(1+1))$
124	H(Z) = PAC(J) + AC(J) + CA(J) + CA(J
122	H(J) = PPA(J) + A(J) + C T A(J)
120	H(4) = PAC(N) + AC(1) + SIN(AC(1) K(1) + C(1) + C
121	H(1+4) = 1((1+1) + ((1)+(0)) A L(1+1) + A C(1) + S L(A L(1+1))
128	
129	$\prod_{i=1}^{n} \sum_{j=1}^{n} \prod_{i=1}^{n} \sum_{j=1}^{n} \prod_{i=1}^{n} \prod_{j=1}^{n} \prod_{i$
120	
133	$\prod_{i=1}^{n} \prod_{j=1}^{n} \prod_{i=1}^{n} \prod_{i=1}^{n} \prod_{j=1}^{n} \prod_{i=1}^{n} \prod_{j=1}^{n} \prod_{i=1}^{n} \prod_{i=1}^{n} \prod_{j=1}^{n} \prod_{i$
122	
134	→ 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
126	
136	= -2 X (K) + C (X + C (X + 1 +
127	
130	$\mu(\tau + 1) = -\mu(\tau + \lambda) \pm (\pi(\tau + 1)) \pm (\pi(\tau + 1)) + \mu(\tau + 1) + \mu(\tau + 1))$
130	H(14) = PXC(1) + STV(A(3)(A(3)) - PYC(1) + C(A(3)) + XC(A(3)) + XC(A(3)) + YC(A(3)) +
140	\$) * \$T N (A 1 (J))
141	H(15) = PPXC(J) + STN(AL1(J)) - PPYC(J) + COS(AL1(J)) + 2.0 + PXC(J) + COS(AL1(J))
142	\$) 1+2, 0* PYC(J)*S IN(AL1(J))-X C(J)*S IN(AL1(J))+Y C(J)*COS(AL1(J))
143	H(16) = PXC(K) + SIN(AL1(K)) - PYC(K) + COS(AL1(K)) + XC(K) + COS(AL1(K)) + YC(K)
144	\$) *\$ TN (AL1 (K))
145	DO 72 I = 1.16
146	72 H1(1)=H(1)
147	DO 74 $I = 1, 16$
148	74 H2(I) = H(I)
149	1=1
150	R(I)=0.5*(H(I)*H(I)+H(I+4)*H(I+4))
151	R(2)=(YC(1)*XC(J)-XC(1)*YC(J)-YC(1)*PYC(J)-XC(1)*PXC(J))*COS(AL1(J
152	\$}}+(XC(1)*XC(J)+YC(1)*YC(J}-XC(1)*PYC(J)+YC(1)*PXC(J)}*SIN(AL1(J))
153	\$+XC(J)*PXC(J)+YC(J)*O
154	ZS1=2.0*YC(1)*PXC(J)-2.0*XC(1)*PYC(J)-YC(1)*PPYC(J)-XC(1)*PPXC(J)+
155	\$XC(1)*XC(J)+YC(1)*YC(J)
156	Z S2=2•0*XC(1)*PXC(J)+2•0*YC(1)*PYC(J)-XC(1)*PPYC(J)+YC(1)*PPXC(J)-
157	\$YC(1)*XC(J)+XC(1)*YC(J)
158	R(3)=COS(AL1(J))*ZS1+SIN(AL1(J))*ZS2+XC(J)*PPXC(J}+YC(J)*PPYC(J)+P
159	\$XC(J)*PXC(J)+PYC(J)*PYC(J)
160	R(4)=(YC(1)*XC(K)-XC(1)*YC(K)-YC(1)*PYC(K)-XC(1)*PXC(K))*COS(AL1(K
161	\$)]+(XC(1)*XC(K)+YC(1]*YC(K)-XC(1]*PYC(K)+YC(1]*PXC(K))*SIN(AL1(K))
162	\$+XC(K)*PXC(K)+YC(K)*PYC(K)

CARD 163 P(I) = 1.0 - COS(AL1(I+1))P(2)=SIN(AL1(J)) 164 P(3)=COS(AL1(J)) 165 P(4)=SIN(AL1(K)) 166 167 Q(I)=SIN(AL1(I+1)) 168 Q(2)=COS(AL1(J)) Q(3) = -SIN(AL1(J))169 170 Q(4) = COS(AL1(K)) 171 CALL SIMQ(H,R,4,KS) CALL SIMQ(H1,P,4,KS1) 172 173 CALL SIMQ(H2,Q,4,KS2) 174 WRITE (6,99) KS 175 WRITE (6,99) KS1 176 WRITE (6,99) KS2 99 FORMAT (1X, 11) 177 F1=Q(1)*Q(3)+Q(2)*Q(4) 178 179 $F_{2}=P(1)*Q(3)+Q(1)*P(3)+P(2)*Q(4)+Q(2)*P(4)$ 180 $F_{3=R(1)*Q(3)+Q(1)*R(3)+Q(2)*R(4)+Q(4)*R(2)$ 181 F4=P(1)*P(3)+P(2)*P(4)F = P(1) * R(3) + P(3) * R(1) + P(2) * R(4) + P(4) * R(2) - 1.0182 F6=R(1)*R(3)+R(2)*R(4) 183 184 G1=Q(1)*Q(4)-Q(2)*Q(3) $G_{2=P(1)*Q(4)+Q(1)*P(4)-P(2)*Q(3)-Q(2)*P(3)$ 185 G3=R(1)*Q(4)+Q(1)*R(4)-R(2)*Q(3)-Q(2)*R(3)-1.C 186 G4=P(1)*P(4)-P(2)*P(3)187 188 G5=P(1)*R(4)+R(1)*P(4)+P(2)*R(3)-P(3)*R(2) 189 G6=R(1)*R(4)-R(2)*R(3)190 XCOF(1)=F1*F1*G6*G6+G3*G3*F1*F6-G3*G6*F1*F3+G1*G6*F3*F3+G1*G1*F6*F 191 \$6-G1*G3*F3*F6-2.0*G1*G6*F1*F6 192 XCOF(2)=2.0+G5+G6+F1+F1+2.0+G2+G3+F1+F6+G3+G3+F1+F5-G3+G6+F1+F2-G2 193 \$*G6*F1*F3-G3*G5*F1*F3+2.0*G1*G6*F2*F3+G1*G5*F3+F3+2.0*G1*G1*F5*F6-194 \$G1*G3*F2*F6-G1*G2*F3*F6-G1*G3*F3*F5-2.0*G1*G5*F1*F6-2.0*G1*G6*F1*F 195 \$5 XCDF(3)=2.0*G4*G6*F1*F1+G5*G5*F1*F1+G2*G2*F1*F6+2.0*G2*G3*F1*F5+G3 196 \$*G3*F1*F4~G2*G6*F1*F2-G3*G5*F1*F2~G2*G5*F1*F3-G3*G4*F1*F3+G1*G6*F2 197 \$*F2+2.0*G1*G5*F2*F3+G1*G4*F3*F3+2.0*G1*G1*F4*F6+G1*G1*F5*F5-G1*G2* 198 \$F2*F6-G1*G3*F2*F5-G1*G2*F3*F5-G1*G3*F3*F4-2.0*G1*G6*F1*F4-2.0*G1*G 199 200 \$5*F1*F5-2.0*G1*G4*F1*F6 XCDF(4)=2.0+G4+G5+F1+F1+G2+G2+F1+F5+2.0+G2+G3+F1+F4-G2+G5+F1+F2-G3 201 \$*64 *F1 * F2 - 62 * 64 *F1 *F3 + 61 *65 *F2 *F2 + 2 • 0 *6 1 * 64 *F2 *F3 + 2 • 0 * 61 * 61 * F4 * F5 -202 \$G1*G2*F2*F5-G1*G3*F2*F4-G1*G2*F3*F4-2.0*G1*G5*F1*F4-2.0*G1*G4*F1*F 203 204 \$5 XCOF(5)=G4*G4*F1*F1+G2*G2*F1*F4-G2*G4*F1*F2+G1*G4*F2*F2*F2*F2*F3*F4*F 205 \$4-G 1*G 2*F 2*F 4-2. 0*G 1*G4*F1*F4 206 CALL POLRT (XCOF, COF, 4, ROOTR, ROOTI, IER) 207 208 WRITE (6,109) 209 109 FORMAT (/,27X,9H L1 ROOTS,/) 210 DO 110 I=1,4 110 WRITE (6,112) RCCTR(I), ROOTI(I) 211 112 FORMAT (17X, E13.5, 10X, E13.5, /) 212 WRITE (6,114) IER 213 214 114 FORMAT (1X,11) 215 WRITE (6,106)

74

216

106 FORMAT (1H1,//)

	123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890
CARD	
217	125 DO 215 I=1,4
218	0A=F1
219	DB=F2*RODTR(I)+F3
220	DC=F4*RCOTR(I)*RCOTR(I)+F5*ROOTR(I)+F6
221	OD = G 1
222	$DE = G2 \times R OOTR(1) + G3$
223	OF = G4 * ROOTR(I) * RCOTR(I) + G5 * ROOTR(I) + G6
224	IF (ABS(R00TI(I)).LT.0.000001) ROOTI(I)=0.0
225	IF (R0011(1)) 200,130,200
226	200 WRITE (6,201)
227	201 FORMAT (5X,18H ROOT IS IMAGINARY,7)
228	GO 10 215
229	130 SA=UB+0B-4.0*UA+UC
230	1F(SA) = 200, 140, 140
231	140 ROUTAI = (-08+5)R((5A))/(2+0+0A)
232	
233	
234	
235	
230	
221	WELLS ($0_1 227$) RUCHALINUULAZINUULDINUULDINUULDZ EEDAMAT // 127 AU DOOTAL-ELO (.EV.AU DOOTAL-ELO (/ 127 AU DOOTAL
220	133 FURMAL (7 13A for RUULAL-1 10.073A for RUULAZ-1 FL0.077 (13A for RUULD) t = 10.4 sy ou porto-2 for 4.71
239	
240	
241	16 (ABS(R00TA2-R00TB1)-(1-0.05) G0 T0 170
242	I = (ABS(RO)TA2 - RO)TB2) + I = 0.05) GO TO 170
245	
245	160 WRITE (6.180) REOTAL
246	180 FORMAT (2X, 16H SECOND ROOT IS .F20.5./)
247	ROOT=ROOT A1
248	GO TO 210
249	170 WRITE (6,180) ROOTA2
250	ROOT=ROOT A2
251	210 WRITE (6,212)
252	212 FORMAT (14X,13H CENTER POINT,31X,13H CIRCLE POINT,/,8X,3H XA,19X,3
253	\$H YA,19X,3H XB,19X,3H YB,/)
254	XA(I)=R(1)+P(1)*R00TR(I)+Q(1)*R00T
255	YA(I)=R(2)+P(2)*R00TR(I)+Q(2)*R00T
256	XB(I)=R(3)+P(3)*R00TR(I)+Q(3)*R00T
257	YB(1)=R(4)+P(4)*RODTR(1)+Q(4)*ROOT
258	WRITE (6,230) XA(I),YA(I),XB(I),YB(I)
259	230 FORMAT (4F20.8,/,33X,20H XXXXXXXXXXXXXXXXXX,//)
260	215 CONTINUE
261	STOP
262	END

APPENDIX F

SYNTHESIS PROGRAM FOR TWO FINITE POSITIONS

AND ONE THIRD INFINITESIMAL

DISPLACEMENT

00	00000001111111 34567890123456	11122222222233333333344444444445 7890123456789012345678901234567890	555555555566666666666677 123456789012345678901	777777778 1234567890		
CARD 1 C	****	****	******	*****	•	
2 C 3 C 4 C	* *	SYNTHESIS OF SIX-LINK STEPHENS FUNCTION GENERATOR	ON 2	* * *		
5 C 6 C 7 C	* .	TWO FINITE POSITIONS AND ONE JER	K RATIO	* * *		
8 C 9 C	*	INPUT FINITE ROTATION	TH1(I)	*		
10 C	*	OUTPUT FINITE ROTATION	PH1(1)	* .		
12 C 13 C	*	ASSUMED ROTATION OF TERNARY	AL1(I)	*		
14 C 15 C	*	THIRE INFINITESIMAL DISPLACEMENT	DD DP TH(J)	*		
16 C 17 C	*	OUTPUT LINK LENGTH	QC	* *		
18 C 19 C	*	OUTPLT LINK INITIAL ANGLE	THE	*		
20 C	*	*****	*****	*		
30 31 32 33 34 35 24	DIMENSION READ (5,5) 5 FORMAT (1) WRITE (6,6 6 FORMAT (// READ (5,1) 8 FORMAT (//	PXC(5),PYC(5),PPXC(5),PPYC(5),PPPX)))) J (15X,31H JERK RATIO SPECIFIED AT P) TH1(2)) PH1(2)	C(5),PPPYC(5) OINT ,11,//)			
30 37 38	READ (5,10 10 FORMAT (F))) AL1(2) 0.0)	•			
39 40 41 42	READ (5,20 READ (5,20 READ (5,20 READ (5,20 READ (5,20)) QC,THE)) DPHCTH(J),CTHDAL(J))) DDPHTH(J),DDTHAL(J))) DDDPTH(J),ODDTAL(J)				
43 44 45 46	20 FORMAT (25 WRITE (6,1 11 FORMAT (13 WRITE (6,1	10.0) 1) TH1(2) X,24H INPUT ROTATION ANGLE IS,2X,F 2) PH1(2)	10.3,/)			
47 48 49	12 FORMAT (13 WRITE (6,1 13 FORMAT (13	BX,25H OUTPUT ROTATION ANGLE IS,1X, 3} AL1(2) BX,22H TERNARY LINK ROTATION,4X,F10	F10.3,/)			
50 51 52	WRITE (6,1 14 FORMAT (4x \$0.3.7/)	4) QC,THE ,15H CUTPUT LINK IS,F10-3,24H LONG	AND AT AN ANGLE OF F	-1		
26						

	123456789012345678901234567890123456789012345678901234567890123456789012345678901
CAPD	12343616361234367670123436163612343616361234361636123436163612343616361234361636123436163612343616361234361636
55	WRITE (6.16) DDENTH(1), DDTHAL(1)
56	16 COPMAT (72.11H DODHTH(1)=.610.3.10X.11H DOTHAL(1)=.610.3.()
57	WRITE (6.17) DDPTH(1), DDPTH(1)
58	17 FORMAT (7X-11) FORDEH() = FO.3-10X-11H DDDTA(() = FO.3-//)
59	
60	YO(1)=0.0
61	XM=0.0
62	YM= 9 - 9
63	TH1 (1)=0.0
64	PH1(1) = 0.0
65	A_{1} (1) = 0 - 0
66	THE=(THE*3.14159265)/180.0
67	I = 2
68	TH1(1)=(TH1(1)*3.14159265)/180.0
69	PH1(I)=(PH1(I)*3.14159265)/180.0
70	AL1(I)=(AL1(I)*3.14159265)/180.0
71	XC(1)=1.0+QC+CDS(THE)
72	YC(1)=0.0+CC*SIN(THE)
73	XQ(I)=1.0*COS(-TH1(I))
74	YQ(1)=1.0*SIN(-T+1(1))
75	XC(I)=XC(1)*COS(PH1(I)-TH1(I))+YC(1)*SIN(PH1(I)-TH1(I))+XQ(I)-XQ(1
76	<pre>\$)*COS(PH1(I)-TH1(I))+YQ(1)*SIN(PH1(I)-TH1(I))</pre>
77	YC(I)=XC(1)*SIN(PH1(I)-TH1(I))+YC(1)*COS(PH1(I)-TH1(I))+YQ(I)-XQ(1
78	\$)*SIN(PH1(I)-TH1(I))-YQ(1)*COS(PH1(I)-TH1(I))
79	DXC(J)=(XC(1)-XQ(1))*(SIN(PH1(J)-TH1(J))-SIN(PH1(J)-TH1(J))*DPHDTH
80	\$(J))+(YC(1)-YQ(1))*(COS(PH1(J)-TH1(J))-COS(PH1(J)-TH1(J))*DPHDTH(J
81	\$))+XQ(1)*SIN(-TH1(J))+YQ(1)*COS(-TH1(J))
82	DYC(J)=(XC(1)-XC(1))*(-COS(PH1(J)-TH1(J))+COS(PH1(J)-TH1(J))*DPHDT
83	\$H(J}}+(YC(1)-YQ(1))*(SIN(PH1(J)-TH1(J))-SIN(PH1(J)-TH1(J)}*DPHDTH(
84	\$J })-XQ(1) *COS(-TH1(J} }+YQ(1) *SIN(-TH1(J) }
85	DDXC(J)=(XC(1)-XQ(1))*(-COS(PH1(J)-TH1(J))+2.0*COS(PH1(J)-TH1(J))*
86	\$DPHDTH(J)-COS(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-TH1(J)
87	\$)*DDPHTH(J))+(YC(1)-YQ(1))*(SIN(PH1(J)-TH1(J))-2+0*SIN(PH1(J)-TH1(
88	\$J})*DPHDTH(J)+SIN(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-COS(PH1(J)-TH
89	\$1(J))*DDPHTH(J))-XQ(1)*COS(-TH1(J))+YQ(1)*SIN(-TH1(J))
90	DDYC(J)=(XC(1)-XQ(1))*(-SIN(PH1(J)-TH1(J))+2.0*SIN(PH1(J)-TH1(J))*
91	\$DPHDTH(J) - SIN(PH1(J) - TH1(J)) * DPHDTH(J) * DPHDTH(J) + COS(PH1(J) - TH1(J)
92	\$)*DDP H1H(J)}+(YC(1)-YQ(1))*(-COS(PH1(J)-TH1(J))+2.0*COS(PH1(J)-TH1
93	\$(J) }*DPHDTH(J)-CGS(PH1(J)-1H1(J))*DPHDTH(J)*DPHD1H(J)-SIN(PH1(J)-T
94	f(J) = 0 Print (J) = x ((I) = SIN(-IHI(J)) = Y ((I) = CUS(-THI(J))
95	DDD I=-S IN (PHI(J) - HI(J)) + 3 U = SIN (PHI(J) - HI(J) = DDI H(J) - 3 U = SIN (PHI(J) - 3 U = SIN (PHI
96	\$PHI(J)-1HI(J) #CPHDIH(J) #CPHDIH(J)+3-0#CUS(PHI(J)-1HI(J))#DDPHIH(J
97	\$1-3.0*UUS(PH1(3)-1H1(3)1*DPHDH(4)31*DPHH(3)+51N(PH1(3)-1H1(3))*DP
98	
99	
100	
101	\$) +2 +0 +2 10 (PTI (J) +1 HI (J) + UPHU H (J) +2 UPHI H (J) + UP
102	$\frac{1}{2} \frac{1}{2} \frac{1}$
103	2003-003(FT1(3)-7)(1(3)/-2)(0)(0)(1)12(3)(1)(1)(1)(1)(1)(1)(3)(2)(0)(1)(1)(1)(1)(1)(1)(1)(1)(1)(1)(1)(1)(1)
104	יון ארע אין ארע אין ארע ארע גער גער גערארע אין ארטארע אין ארע ארע אין ארע גער גערע גער גער גער גער גער גער גער ארע גער גער גער גער גער גער גער גער גער ג
105	
107	
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100	ALTITATE THAT AT ALAR ALAR HIGH HIGH ALAR HIGH ALAR ALAR ALAR ALAR ALAR ALAR ALAR ALA

~

CARD 109 \$)-3.0*CGS(PH1(J)-TH1(J)*DPHDTH(J)*DPHTH(J)+SIN(PH1(J)-TH1(J))*DP 110 \$HOT H(J)*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-TH1(J))*DDDPTH(J) DDDXC(J) = (XC(1) - XQ(1)) * DDD1 + (YC(1) - YQ(1)) * DDD2 - XQ(1) * SIN(-TH1(J)) - (YC(1)) * DDD2 - XQ(1) * SIN(-TH1(J)) + (YC(1)) * ODD2 - XQ(1) * SIN(-TH1(J)) + (YC(1)) * ODD2 - XQ(1) * SIN(-TH1(J)) + (YC(1)) * (YC(1)) *111 \$YQ(1) *COS(-TH1(J)) 112 DCCYC(J)=(XC(1)-XQ(1))*DDD3+(YC(1)-YQ(1))*DDD4+XQ(1)*COS(-TH1(J))-113 114 \$YQ(1) * SIN(-TH1(J)) WRITE (6,21) DXC(J), DYC(J) 115 21 FORMAT (7X,8H DXC(J)=,3X,F10.3,10X,8H DYC(J)=,3X,F10.3,/) 116 117 WRITE (6.22) DDXC(J),DDYC(J) 22 FORMAT (7X,9H DEXC(J)=,2X,F10.3,10X,9H DDYC(J)=,2X,F1C.3,/) 118 WRITE (6,23) DDCXC(J), DDDYC(J) 119 23 FGRMAT (7X,10H DDDXC(J)=,1X,F10.3,10X,10H DDDYC(J)=,1X,F10.3,//) 120 PXC(J)=DTHDAL(J)*DXC(J) 121 PYC(J)=DTHDAL(J)*DYC(J) 122 PPXC(J)=DDTHAL(J)*DXC(J)+DTHDAL(J)*DTHDAL(J)*DDXC(J) 123 124 PPYC(J)=DDTHAL(J)*DYC(J)+DDTHAL(J)*DDTHAL(J)*DDYC(J) PPPXC(J)=DDDTAL(J)*DXC(J)+3.0*DTHDAL(J)*DDTHAL(J)*DDXC(J)+DTHCAL(J 125 126 \$)*OTHCAL(J)*DTHCAL(J)*DDDXC(J) PPPYC(J)=DDCTAL(J)+CYC(J)+0.0+0+0+(J)+DAL(J)+DCYC(J)+DTHDAL(J) 127 128 \$) *D THDAL(J) *DTHDAL(J) *DDDYC(J) 129 WRITE (6,31) 31 FORMAT (11X, 3H XQ, 18X, 3H YQ, 18X, 3H XC, 18X, 3H YC, /) 130 131 DD 33 I=1,2 33 WRITE (6,32) XQ(1), YQ(1), XC(1), YC(1) 132 32 FORMAT (6X,F12.8,9X,F12.8,9X,F12.8,9X,F12.8,/) 133 134 I=1 135 H(I)=XC(I+1)-XC(1)*COS(AL1(I+1))+YC(1)*SIN(AL1(I+1)) H{2}=PXC(J)+XC(1)*SIN(AL1(J))+YC(1)*COS(AL1(J)) 136 H(3)=PPXC(J)+XC(1)*COS(AL1(J))-VC(1)*SIN(AL1(J)) 137 H(4) = PPPXC(J) - XC(1) * SIN(AL1(J)) - YC(1) * CCS(AL1(J)) 138 H(I+4)=YC(I+1)-YC(1)*COS(AL1(I+1))-XC(1)*SIN(AL1(I+1)) 139 H(6)=PYC(J)-XC(1)*COS(AL1(J))+YC(1)*SIN(AL1(J)) 140 H(7) = PPYC(J) + XC(1) * SIN(AL1(J)) + YC(1) * COS(AL1(J)) 141 H(8)=PPPYC(J)+XC(1)*COS(AL1(J))-YC(1)*SIN(AL1(J)) 142 143 H(I+8) = -H(I) * COS(AL1(I+1)) - H(I+4) * SIN(AL1(I+1))H(10) = -PXC(J) * CCS(AL1(J)) - PYC(J) * SIN(AL1(J)) + XC(J) * SIN(AL1(J)) - YC(J) * SIN(AL1(J)) + XC(J) * SIN(AL1(J)) * SIN(AL1144 145 \$J)*COS(AL1(J)) 146 H(11) =- PPXC(J)*COS(AL1(J))-PPYC(J)*SIN(AL1(J))+2.0*PXC(J)*SIN(AL1(147 \$J))-2.0*P YC(J)*COS(AL1(J))+XC(J)*COS(AL1(J))+YC(J)*S IN(AL1(J)) H(12)=-PPPXC(J)*COS(AL1(J))-PPPYC(J)*SIN(AL1(J))+3.G*PPXC(J)*SIN(A 148 \$L1(J))-3.0*PPYC(J)*CCS(AL1(J))+3.0*PXC(J)*COS(AL1(J))+3.0*PYC(J)*S 149 \$IN(AL1(J))-XC(J)*SIN(AL1(J))+YC(J)*COS(AL1(J)) 150 H(I+12) =- H(I+4) *COS (AL1(I+1)) +H(I) *SIN(AL1(I+1)) 151 H(14)=PXC(J)*SIN(AL1(J))-PYC(J)*COS(AL1(J))+XC(J)*COS(AL1(J))+YC(J 152 \$)*SIN(AL1(J)) 153 H(15) = PPXC(J)*SIN(AL1(J))-PPYC(J)*COS(AL1(J))+2.0*PXC(J)*COS(AL1(J 154 \$))+2.0*PYC(J)*SIN(AL1(J))-XC(J)*SIN(AL1(J))+YC(J)*COS(AL1(J)) 155 H(16)=PPPXC(J)*SIN(AL1(J))-PPPYC(J)*COS(AL1(J))+3.0*PPXC(J)*COS(AL 156 \$1(J)+3.0*PPYC(J)*SIN(AL1(J))-3.0*PXC(J)*SIN(AL1(J))+3.0*PYC(J)*C0 157 \$S(AL1(J))-XC(J)*COS(AL1(J))-YC(J)*SIN(AL1(J)) 158 00 72 I=1,16 159 72 H1(I)=H(I) 160 DO 74 I≠1.16 161 162 74 H2(I) = H(I)

CARD		
163	I =1	
164	R(I)=0.5*(H(I)*H(I)+H(I+4)*H(I+4)}	
165	R(2)=(YC(1)*XC(J)-XC(1)*YC(J)-YC(1)*PYC(J)-XC(1)*PXC(J))*COS(AL1(J	
166	\$))+(XC(1)*XC(J)+YC(1)*YC(J)-XC(1)*PYC(J)+YC(1)*PXC(J))*SIN(AL1(J))	
167	\$+XC(J)*PXC(J)+YC(J)	
168	$DD1 = 2 \cdot 0 * Y C(1) * P X C(1) = 2 \cdot 0 * X C(1) * P Y C(1) = Y C(1) * P P Y C(1) * P P X C(1) + 0$	
140		
170		
171	$\frac{1}{1+1} \frac{1}{1+1} \frac{1}$	
1/1		
112	R(3) = COS(ALI(3)) + DOI + SIN(ALI(3)) + DO2 + AC(3) + PPAC(3) +	
1/3	\$X((J)*PX((J)*PY((J)*PY((J)	
174	$DDD5 = -YC(1) * PPPYC(J) - XC(1) * PPPXC(J) + 3 \cdot 0 * YC(1) * PPXC(J) - 3 \cdot 0 * XC(1) * PP$	
175	\$YC(J)+3.0*XC(1)*PXC(J)+3.0*YC(1)*PYC(J)-YC(1)*XC(J)+XC(J)*YC(J)	
176	DDD6=-XC(1)*PPPYC(J)+YC(1)*PPPXC(J)+3.0*XC(1)*PPXC(J)+3.0*YC(1)*PP	
177	\$YC(J)-3.0*YC(1)*PXC(J)+3.0*XC(1)*PYC(J)-XC(1)*XC(J)-YC(1)*YC(J)	
178	R(4)=C3S(AL1(J))*DDD5+SIN(AL1(J))*DDD6+XC(J)*PPPXC(J)+YC(J)*PPPYC(
179	\$j}+3.0*PXC(J)*PPXC(J)+3.0*PYC(J)*PYC(J)	
180	P(I)=1.0-COS(AL1(I+1))	
181	P(2) = SIN(ALI(J))	
1.82	P(3) = COS(A(1), 1)	
183	P(4) = -SIN(4)(1,1)	
194		
104		
105		
100		
187		
188	CALL SIMUCH, K, 4, KSJ	
189	CALL SIMO(H1,P,4,KSI)	
190	CALL SIMQ(H2, C, 4, KS2)	
191	WRITE (6,99) KS	
192	WRITE (6,99) KS1	
193	WRITE (6,99) KS2	
194	99 FORMAT (1X,I1)	
195	F1=Q(1)+Q(3)+Q(2)+Q(4)	
196	F2=P(1)*Q(3)+Q(1)*P(3)+P(2)*Q(4)+Q(2)*P(4)	
197	$F_{3=R}(1) * O(3) + O(1) * R(3) + O(2) * R(4) + O(4) * R(2)$	
108	$F_{4} = P(1) * P(3) + P(2) * P(4)$	
100	$5 = 0(1) \times 0(3) + 0(3) \times 0(1) + 0(2) \times 0(4) \times 0(4) \times 0(2) = 1.0$	
200		
200		
201	01-4(1)-4(7)-4(2)-4(3)- co-01(1)-4(7)-4(2)-0(3)+0(3)-0(2)+0(3)	
202		
203	$G_{3}=R(1)*Q(4)+Q(1)*R(4)-R(2)*Q(3)-Q(2)*R(3)-1.0$	
204	G4=P(1) + P(4) - P(2) + P(3)	
205	$G_{5=P[1]*R(4)+R(1)*P(4)-P(2)*R(3)-P(3)*R(2)$	
206	G6=R(1)*R(4)-R(2)*R(3)	
207	XCQF(1)=F1*F1*G6*G6+G3*G3*F1*F6-G3*G6*F1*F3+G1*G6*F3*F3+G1*G1*F6*F	
208	\$6-G1*G3*F3*F6-2.0*G1*G6*F1*F6	
209	XCDF{2}=2.0*G5*C6*F1*F1+2.0*G2*G3*F1*F6+G3*G3*F1*F5-G3*G6*F1*F2-G2	
210	\$*G6*F1*F3-G3*G5*F1*F3+2.0*G1*G6*F2*F3+G1*G5*F3+F3+2.0*G1*G1*F5*F6-	
211	\$G1*G3*F2*F6~G1*G2*F3*F6-G1*G3*F3*F5-2.0*G1*G5*F1*F6-2.0*G1*G6*F1*F	
212	\$5	
213	XCDF(3)=2.0*G4*G6*F1*F1+G5*G5*F1*F1+G2*G2*F1*F6+2.0*G2*F1*F6+63	
214		
215		
212	\$F\$\L_{__________________	
210	₽F ፈ∻ F O= 5 1∻5 3 *F ፈ∻ F J= 5 1∻ 5∠*F J≠F J=5 1*5 3*F J≠ F 4=4 •V* 51* 60* F1*F4+2 •V*61*6	

12345678901234567890123456789012345678901234567890123456789012345678901234567890123456789012345678901234567890 CARD \$5*F1*F5-2.0*G1*G4*F1*F6 217 XCOF(4)=2_0*G4*G5*F1*F1+G2*G2*F1*F5+2.0*G2*G3*F1*F4-G2*G5*F1*F2-G3 218 \$*G4*F1*F2-G2*G4*F1*F3+G1*G5*F2*F2+2.0*G1*G4*F2*F3+2.0*G1*G1*G1*F4*F5-219 \$G1*G2*F2*F5-G1*G3*F2*F4-G1*G2*F3*F4-2.0*G1*G5*F1*F4-2.0*G1*G4*F1*F 220 \$5 221 222 XCOF(5)=G4*G4*F1*F1+G2*G2*F1*F4-G2*G4*F1*F2+G1*G4*F2*F2+G1*G1*G1*F4*F \$4-G1*G2*F2*F4-2.0*G1*G4*F1*F4 223 CALL POLRT (XCCF, COF, 4, ROOTR, ROOTI, IER) 224 225 WRITE (6,109) 109 FORMAT (/,27X,9H L1 RODTS,/) 226 227 DO 110 I=1.4 110 WRITE (6,112) ROOTR(I), ROOTI(I) 228 112 FORMAT (17X, E13.5, 10X, E13.5,/) 229 230 WRITE (6,114) IER 114 FORMAT (1X, I1) 231 WRITE (6,106) 232 1C6 FORMAT (1H1,//) 233 125 DO 215 I=1,4 234 0A=F1 235 OB = F2 + ROOTR(I) + F3236 OC=F4*ROOTR(I)*ROOTR(I)*F5*ROOTR(I)+F6237 238 0D=G1 DE=G2*RODTR(I)+G3239 OF=G4*ROOTR(I)*ROOTR(I)+G5*ROOTR(I)+G6 240 IF (ABS(ROOTI(I)).LT.0.000001) ROOTI(I)=0.0 241 IF (ROOTI(I)) 200,130,200 242 200 WRITE (6,201) 243 201 FORMAT (5X,18H ROOT IS IMAGINARY,/) 244 245 GO TO 215 130 SA=08*08-4.0*0A*CC 246 247 IF (SA) 200,140,140 248 140 ROOTA1= (-OB+SQRT(SA))/(2.0*OA) 249 RODTA2= (-08-SQRT (SA))/(2.0+0A) 250 SB=0E*0E-4.0*0D*0F 251 IF (SB) 200,150,150 150 ROOTB1=(-OE+SQRT(SB))/(2.0*0D) 252 ROOTB2= (-DE-SQRT(SB)) /(2.0+0D) 253 WRITE (6,155) RCCTA1, ROOTA2, ROOTB1, ROOTB2 254 155 FORMAT (/,13X,8H ROOTA1=,F18.6,5X,8H ROOTA2=, F18.6,/,13X,8H ROOTB1 255 \$=, F18.6, 5X, 8H ROOTB2=, F18.6, /) 256 IF (ABS (ROOT A1-ROOT B1).LT.0.05) GO TO 160 257 IF (ABS(ROOTA1-ROOTB2).LT.0.05) GO TO 160 258 IF (ABS(ROOTA2-ROOTB1).LT.0.05) GO TO 170 259 IF (ABS(ROUTA2-ROOTB2).LT.0.05) GO TO 170 260 -GO TO 215 261 160 WRITE (6,180) ROOTA1 **2**62 180 FORMAT (2X,16H SECOND ROOT IS ,F20.5,/) 263 ROOT=ROOTA1 264 GO TO 210 265 170 WRITE (6,180) RCCTA2 266 ROOT = ROOTA2267 210 WRITE (6,212) 268 212 FORMAT (14X,13H CENTER POINT,31X,13H CIRCLE POINT,/,8X,3H XA,19X,3 269 \$H YA, 19X, 3H XB, 19X, 3H YB, /} 270

L8

CARD 271

XA(I)=R(1)+P(1)*ROOTR(I)+Q(1)*ROOT

272 YA(I)=R(2)+P(2)*ROOTR(I)+Q(2)*ROOT

273 XB(I)=R(3)+P(3)*ROOTR(I)+Q(3)*ROOT

274 YB(I)=R(4)+P(4)*ROOTR(I)+Q(4)*ROOT

275 WRITE (6,230) XA(I),YA(I),XB(I),YB(I)

276 230 FORMAT (4F20.8,/,33X,20H XXXXXXXXXXXXXXXXXXX//)

277 215 CONTINUE

278 STOP

279 END

APPENDIX G

SYNTHESIS PROGRAM FOR ONE FINITE POSITION

AND ONE FOURTH INFINITESIMAL

DISPLACEMENT

** ** ***	*****	*****	*****	*******	*****
* `					
*	SYN	THESIS	OF SIX-	LINK STEPH	IENSON 2
*	•	F	UNCTION	GENERATOR	
*					
*	ON E	FINITE	POSITIO	N AND ONE	KERK RATIO
*					•
*					
*	FOURTH	INFIN	ILLESIMAL	DISPLACE	MENT DODOPT(J)
*	0.07011	1 7 814		•	00
*	001001	LINK	LENGIN		
*	OUTPUT	TTNK	INTTIAL	ANGLE	THE
*					
*****	*****	*****	********	*******	*****
DIM	NSION XCOF(5)	+C-OF(5	5),RODTR(4),ROOTI(4	4)
DIM	NSION XQ(5),Y	Q(5),X	(A(5),YA(5), XB(5), Y	(B(5),XC(5),YC(5)
DIM	NSION THI(5)	PH1(5)	,ALI(5)		
DIM	NSION H(16), H	1(16),	H2(16),R	(4),P(4),G	1(4)
DIM	NSICN DPHDTH	5), CCP	PHTH(5),D	DDPTH(5),C	DDDDPT(5)
DIME	NSION DTHDAL	5),DDT	HAL(5),D	DDTAL(5),	DDDDTA(5)
DIM	NSION DXC(5),	DYC(5)	DDXC(5)	DDYC(5)	
DIM	NSIUN DUDACIS	1,0001			5646(5)
DIM	NSION PACIDIA	1 0004	JPF X6 (5)	17716(3) DVC(5) DDC	
1-1	INSION PPPACES	1, PPP1	CLOTTERE	PACIDINPPP	FIC(5)
	E (6.6)				,
6 508	AT (//.12X.32	H KERK	RATIO S	PECIFIED A	AT POINT 1.//)
REAL	(5.20) QC.TH	Ε			
REA	(5,20) DPHCT		THDAL(J)		
REAL	(5,20) DDPH1	H(J),D	DTHAL(J)		
REAL	(5,20) DDDPT	H(J) ,D	DDTAL(J)		
REA	(5,20) DDDCF	T(J),C	DDDTA(J)		
20 FOR	AT (2F10.C)				
WRI	E (6,14) QC,1	HE			
14 FOR	AT (4X,15H CU	TPUT L	INK IS, F	10.3,24H L	UNG AND AT AN ANGLE OF,F1
\$0.3	//)				
15 FOR	A: (/X, 115 EP	HU1H(J	DTUDAL 1	TOX TIH D	JIHUAL(J) =, F1C. 3,/)
WKI WKI	E (0+15) UPPL	1707 H	DI HUAL (J		
16 500	C (01101 UUPP		14 E10 2	10¥.11⊨ ⊓	DTHAL (1)- 510 3. ()
TO LOKI	F (6.17) 0000	TH (.1)) ION9 IIN L	//////////////////////////////////////
17 500	AT (7X-11H DD	DPTH	1=.F10.3	10X.11H E	$DDTAL(1) = EI 0.3 \cdot / I$
WRT	E (6.18) DDDD	PT (.) .	DODDTALL)	
18 FOR	AT (7X.11H DC	DDPT)=.F10.3	-10X-11H	DDDDTA(J) = .F10.3.//)
XQL)=1.0				
YOI)=0.0				
XM= (• 0				
Y M=(.0				
TH1	1)=0.0				
(PH1)	1)=0.0				
AL1	1 = 0.0				
THE	(THE* 3.141592	65)/18	0.0		

CARD XC(1)=1.0+QC*COS(THE) 55 YC(1)=0.0+QC*SIN(THE) 56 DXC(J) = (XC(1) - XC(1)) * (SIN(PH1(J) - TH1(J)) - SIN(PH1(J) - TH1(J)) * DPHDTH57 \$(J))+(YC(1)-YQ(1))*(CCS(PH1(J)-TH1(J))-COS(PH1(J)-TH1(J))*DPHDTH(J 58 59 \$))+XQ(1)*SIN(-TH1(J))+YQ(1)*COS(-TH1(J)) 60 DYC(J) = (XC(1) - XC(1)) * (-COS(PH1(J) - TH1(J)) + COS(PH1(J) - TH1(J)) * DPHDT\$H(J) + (YC(1) - YQ(1)) * (SIN(PH1(J) - TH1(J)) - SIN(PH1(J) - TH1(J)) * DPHDTH(61 62 \$J))-XQ(1)*COS(-T+1(J))+YQ(1)*SIN(-TH1(J)) DDXC(J)=(XC(1)-XG(1))*(-COS(PH1(J)-TH1(J))+2-0*COS(PH1(J)-TH1(J))* 63 \$DPHDTH(J)-COS(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-TH1(J) 64 \$)*DDPHTH(J))+(YC(1)-YQ(1))*(SIN(PH1(J)-TH1(J))-2.0*SIN(PH1(J)-TH1(65 \$J))*DPHDTH(J)+SIN(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-COS(PH1(J)-TH 66 67 \$1(J) *DDP HT H(J))-XQ(1)*COS(-TH1(J))+YQ(1)*SIN(-TH1(J)) DDYC(J)=(XC(1)-XG(1))*(-SIN(PH1(J)-TH1(J))+2.0*SIN(PH1(J)-TH1(J))* 68 sDPHDTH(J)-SIN(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)+COS(PH1(J)-TH1(J) 69 \$)*DDPHTH(J))+(YC(1)-YQ(1))*(-C0'S(PH1(J)-TH1(J))+2.0*C0S(PH1(J)-TH1 70 \$(J))*DPHDTH(J)-COS(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)-SIN(PH1(J)-T 71 \$H1(J))*DDPHTH(J))-XQ(1)*SIN(-TH1(J))-YQ(1)*COS(-TH1(J)) 72 73 DDD1=-SIN(PH1(J)-TH1(J))+3.0*SIN(PH1(J)-TH1(J))*DPHDTH(J)-3.0*SIN(\$PH1(J)-TH1(J)}*DPHDTH(J)*DPHDTH(J)+3.0*COS(PH1(J)+TH1(J))*DPPHTH(J 74 \$}-3.0*CDS(PH1(J)-TH1(J))*DPHDTH(J)*DDPHTH(J)+SIN(PH1(J)-TH1(J))*DP 75 76 \$HDTH(J)*DPFDTF(J)*DPHDTH(J)-SIN(PHI(J)-THI(J))*DDDPTH(J) DDD2=-COS(PH1(J)+TH1(J))+3.0*COS(PH1(J)-TH1(J))*DPHDTH(J)-3.0*COS(77 \$PH1(J)-TH1(J) }*DPHDTH(J)*DPHDTH(J)-3.0* SIN(PH1(J)-TH1(J))*DDPHTH(J 78 \$)+3.0*SIN(PH1(J)-TH1(J))*DPHDTH(J)*DDPHTH(J)+COS(PH1(J)-TH1(J))*DP 79 \$HDTH(J)*DPHDTH(J)*DPHDTH(J)-COS(PHL(J)-THL(J))*DDDPTH(J) 80 DDD3=COS(PH1(J)-TH1(J))-3.0*COS(PH1(J)-TH1(J))*DPHDTH(J)+3.0*COS(P 81 \$H1(J)-TH1(J) #DPHDTH(J) *DPHDTH(J) +3 .0 *S IN(PH1(J)-TH1(J))*DDPHTH(J) 82 \$-3.0*SIN(PH1(J)-TH1(J))*DPHDTH(J)*DDPHTH(J)-COS(PH1(J)-TH1(J))*DPH 83 \$DTH(J)*DPHDTH(J)*DPHDTH(J)+COS(PH1(J)-TH1(J))*DDDPTH(J) 84 85 DDD4=-SIN(PH1(J)-TH1(J))+3.0*SIN(PH1(J)-TH1(J))*DPHDTH(J)-3.0*SIN(86 \$PH1(J)-TH1(J) *DPHDTH(J)*DPHDTH(J)+3.0*C0S(PH1(J)-TH1(J))*DDPHTH(J 87 \$)-3.0*COS(PH1(J)-TH1(J))*DPHDTH(J)*DDPHTH(J)+SIN(PH1(J)-TH1(J))*DP 88 \$HDTH(J)*DPHDTH(J)*DPHDTH(J)-SIN(PHI(J)-THI(J))*DDDPTH(J) 89 DDCXC(J)=(XC(1)-XQ(1))*DDD1+(YC(1)-YQ(1))*DDD2-XQ(1)*SIN(-TH1(J))-90 \$YQ(1)*COS(-TH1(J)) DDD YC (J)= (XC(1)-XQ(1))*DDD3+ (YC(1)-YQ(1))*DDD4+XQ(1)*COS(-TH1(J))-91 92 **\$YQ(1)***SIN(-TH1(J)) DDDD1=COS(PH1(J)-TH1(J))-4.0*COS(PH1(J)-TH1(J))*DP+DTH(J)+6.0*COS(93 \$PH1(J)-TH1(J)*DPHDTH(J)*DPHDTH(J)+6.0*SIN(PH1(J)-TH1(J))*DDPHTH(J 94 95 \$)-12_0*SIN(PH1(J)-TH1(J))*DPHDTH(J)*DCP+TH(J)+6.0*SIN(PH1(J)-TH1(J s))*DPHDTH(J)*DPHDTH(J)*DDPHTH(J)-4.0*COS(PH1(J)-TH1(J))*DPHDTH(J)* 96 \$DPHDTH(J)*DPHDTH(J)+4.0*COS(PH1(J)-TH1(J))*DDDPTH(J)-4.0*COS(PH1(J) 97 \$) -TH1(J))*DPHDTH(J)*DDPTH(J)-3.0*CDS(PH1(J)-TH1(J))*DDPHTH(J)*DDP 98 \$HTH(J)+COS(PH1(J)-TH1(J))*(DPH0TH(J)**4.0)-SIN(PH1(J)-TH1(J))*DDDD 99 \$PT(J) 100 DDDD2 = -SIN(PH1(J) - T+1(J)) + 4 + 0 + SIN(PH1(J) - TH1(J)) + DPHDTH(J) - 6 + 0 + SIN101 \$(PH1(J)-TH1(J))*DPHDTH(J)*DPHDTH(J)+6.0*COS(PH1(J)-TH1(J))*DPHTH(102 \$J)-12.0*COS(PH1(J)-TH1(J))*DPHDTH(J)*D0PHTH(J)+6.0*COS(PH1(J)-TH1(103 \$J}}*DPHDTH(J)*DPHDTH(J)*DPHTH(J)+0+\$SIN(PH1(J)+TH1(J))*DPHCTH(J) 104 \$*DPHDTH(J)*DPHDTH(J)-4.0*SIN(PH1(J)-TH1(J))*DDDPTH(J)+4.0*SIN(PH1(105 \$ J) --TH1 (J) }* DPHDT+ (J) *DDDPTH(J) +3 .0 *S IN (PH1(J) --TH1(J)) *DDPHTH (J) *DD 106 \$PHTH(J)-SIN(PH1(J)-TH1(J))*(DPHDTH(J)**4.0)-COS(PH1(J)-TH1(J))*DDD 107 \$DPT(J) 108

	12256780012345678001234567880123456780123456788012345678801234567880123456788012345678801234567880123456788012345678801234567880123456788012345678801234567880123456788012345678801234567880123456788012345678801234567880123456788012345888888888888888888888888888888888888
	12343616361234561636123456163612345616361234561636123456163612345616361234561636123456163616361636163616361636666666666666
100	DDDD3=SIN(PH)(J)-TH)(J)-4-0*SIN(PH)(J)-TH)(J)>DDDT3=SIN(DH)(J)+6-0*SIN(
110	(1) - TH(1) + CPHOTH(1) + CPHOTH(1) - (-0 + CHS(PH)(1) - TH(1)) + DOPHTH(1)
111	(1) + 12 = 0 + (0 + 1) + 11 + 11 + 1) + 0 + 0 + 0 + 0 + 0 + 0 + 0 + 0 + 0 +
112	$(1) \times DP = DT = (1) \times DP = CT = (1) \times DDP = T = (1) \times (1) $
112	(1)
114	() = TH (1) + DPHDTH(1) + DDPTH(1) = 3.0*SIN(PH)(1) = TH (1) + DDPHTH(1) + DDP
115	(1) + (1)
116	
117	00004=005(PH1())-TH1())-4,0*0\$(PH1())-TH1())*DPHTH()+6,0*0\$(
118	(1) - TH (1) + DPHDTH(1) + DPHDTH(1) + A - DP + SIN(PH)(1) - TH (1) + DDPHDTH(1)
110	(1 - 1) $(2 - 1)$ $(2 - 1)$ $(3 -$
120	$(1) \times D \cap D$
121	
122	
122	$ = \prod_{i=1}^{n} \prod_{j=1}^{n} \prod$
125	
125	<pre>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>>></pre>
125	
127	$\gamma_{1} = 1 \langle \chi_{1} \rangle - 1 \langle \chi_{1} \rangle + 1 \rangle = 0 \langle \chi_{1} \rangle + 2 \langle \chi_{1} \rangle + $
120	
120	
120	$P_{\mathcal{N}}(\mathbf{x}) = D_{\mathcal{N}} D_{\mathcal{N}}(\mathbf{x}) + D_{\mathcal{N}} D_{\mathcal{N}}(\mathbf{x})$
121	- 10 () - D 110 μ() - D 10 () 10 () - D 10 () 10 () 10 () - D 10 () 10 () - D 10 ()
122	
122	
133	errollar (1)+01HDA((1)+000V(1))
124	
122	PPP IC(3)=000 AC(3)+01C(3)+01C(3)+0C
127	
131	PPPPAC(3) = DDDD(1)A(3) + DA(3) + DA(3) + DDD(1)A(3) + DDD(1)A(3) + DDA(3) + A(3) + DDA(3) + DA(3) + DDA(3) +
130	SUAL (STADUSTAL (STADUAC) STAD. UTIDAL (STADIADAL (STADUAL (STADUAL (STADUAL))
139	
140	PPPPTC(J) = DDDC [A(J) + D[C(J) + 5] (J) + 5] (J) + DC[J] + DC[J] + 1 + DC[J] + DC[J] + 1 + DC[J] + DC[J] + 1 + DC[J] + DC[J] + DC[J] + 1 + DC[J] +
141	
142	S) + D FDAL(J) + D FDAL(J) + D FDAL(J) + D FDAL(J) + D D D U (C(J)
143	$MK1 = \{0, 21\} MK(3) = 3Y = 10 - 3 + 0Y = 0 - 3 + 0Y = 0 - 3 + 0 - 3 $
144	21 FURMAL (13,30 DAC(J) = 33,710,310,00 FUC(J) = 33,710,377
142	WRITE (0,22) DAR(3 , 0) T(3)
140	22 FURMAL (13, 3) DUAC(3) - 323, FLUA 3, FLUA 3, DUIC(3) - 323, DUAC(3) - 323, DUAC(3), DUA
141	$\frac{1}{2} = \frac{1}{2} + \frac{1}$
140	
149	WRITE ($0,24$) UCUUR($1,1,0$) UCUUR($3,1$) 24 FORMAT (72) 15 CODDYC($1,1$) = 102 (102) 15 DDDVC($1,1$) = 102 ($1,1$)
150	$24 \text{ FURMAL } \{1, 1\} \text{ LEUDACUS} \xrightarrow{1} \text{ FUSS} \{10, 5\}, 100, 111 \text{ DUDUC} \{1, -1\}, 100, 500, 100, 111 \text{ DUDUC} \{1, -1\}, 100, 100, 100, 100, 100, 100, 100, 1$
151	WELLS 10,217
152	31 FURMAI (11A,3H XQ,10X,3H TQ,10X,1H AC,10X,3H TC,7/
153	
154	WEITE (0,22) AUTI/(WII)/AUTI/IUT/ 22 FORMAT (/V C12 & OV C12 & OV C12 & V
122	22 FUKMAI (0A,F12.67,74,712.67,74,712.67,74,712.67,74,712.67,74)
156	H(1) = PAC(3) + AC(1) + COC(3) + COC(
157	$H_{\{2\}} = P_{P} A_{\{1\}} A_{\{1\}} F_{C} C_{\{1\}} F_{C} C_{\{1\}} F_{C} C_{\{1\}} F_{C} C_{C} C} C_{C} C_{C} C_{C} C_{C} C_{C} C_{C} C_{C} C} C_{C} C_{C} C_{C} C_{C} C_{C} C} C_{C} C_{C} C C} C_{C} C C C} C C C $
158	H(J) = PPPAL(J) = AL(J) = AL
159	$H(4) = PPPP X_0 (J) - X_0 (J) + COS (AL1 (J) + TO (I) + S (N (AL1 (J)))$
160	$H(\mathbf{J}) = Y \mathbf{U}(\mathbf{J}) - X \mathbf{U}(\mathbf{J}) = \mathbf{U}(\mathbf{J}) + $
161	$H(0) = PPY(1) + X(1) + S IN(\mathbf{A} \perp 1 \mid 1) + Y(1) + C(1) + C(1$
162	H(/)=PPPYC(J)+XC(L)*CUS(ALL(J))-YC(L)*SIN(ALL(J))

CARD $H(8) = PPPPYC(J) - XC(I) \times SIN(AL1(J)) - YC(I) \times CCS(AL1(J))$ 163 164 H(9) = -P XC (J) * COS(AL1(J)) - PYC (J) * SIN(AL1(J)) + XC (J) * SIN(AL1(J)) - YC (J)165 \$)*COS(AL1(J)) H(10) =- PP XC (J) * COS(AL1(J)) - PPYC (J) * SIN(AL1(J)) +2.0 * PX C(J) * SIN(AL1(166 \$J))-2.0*PYC(J)*COS(AL1(J))+XC(J)*COS(AL1(J))+YC(J)*SIN(AL1(J)) 167 H(11) =- PPPXC(J)*COS(AL1(J))-PPPYC(J)*SIN(AL1(J))+3.0*PPXC(J)*SIN(A 168 \$L1(J))-3.0*PPYC(J)*COS(AL1(J))+3.0*PXC(J)*COS(AL1(J))+3.0*PYC(J)*S 169 \$IN(AL1(J))-XC(J)*SIN(AL1(J))+YC(J)*COS(AL1(J)) 170 H(12) =- PPPPXC(J)*COS(AL1(J))-PPPPYC(J)*SIN(AL1(J))+4.0*PPPXC(J)*SI 171 172 \$N(AL1(J))-4.0*PPPYC(J)*COS(AL1(J))+6.0*PPXC(J)*CCS(AL1(J))+6.0*PPY 173 \$C(J)*SIN(AL1(J))-4.0*PXC(J)*SIN(AL1(J))+4.0*PYC(J)*COS(AL1(J))-XC(\$J)*COS(AL1(J))-YC(J)*SIN(AL1(J)) 174 H(13)=PXC(J)*SIN(AL1(J))-PYC(J)*COS(AL1(J))+XC(J)*COS(AL1(J))+YC(J 175 \$) # SIN(AL1(J)) 176 H(14)=PPXC(J)*SIN(AL1(J))-PPYC(J)*COS(AL1(J))+2.0*PXC(J)*COS(AL1(J 177 178 \$))+2.0*PYC(J)*S IN(AL1(J))-XC(J)*S IN(AL1(J))+YC(J)*COS(AL1(J)) 179 H(15)=PPPXC(J)*SIN(AL1(J))-PPPYC(J)*CCS(AL1(J))+3.0*PPXC(J)*COS(AL 180 \$1(J)}+3.0*PPYC(J)*SIN(AL1(J))-3.0*PXC(J)*SIN(AL1(J))+3.0*PYC(J)*CD 181 \$S(AL1(J))-XC(J)*CCS(AL1(J))-YC(J)*SIN(AL1(J)) 182 H(16)=PPPPXC(J)*SIN(AL1(J))-PPPPYC(J)*CCS(AL1(J))+4_0*PPPXC(J)*COS 183 \$(AL1(J))+4.0*PPPYC(J)*SIN(AL1(J))-6.0*PPXC(J)*SIN(AL1(J))+6.0*PPYC \$(J)*COS(AL1(J))-4.0*PXC(J)*CCS(AL1(J))-4.0*PYC(J)*SIN(AL1(J))+XC(J 184 \$)*SIN(AL1(J))-YC(J)*COS(AL1(J)) 185 DO 72 I=1,16 186 72 H1(I) = H(I)187 DO 74 I=1,16 188 74 H2(I)=H(I) 189 R(1)=(YC(1)*XC(J)-XC(1)*YC(J)-YC(1)*PYC(J)-XC(1)*PXC(J))*CDS(AL1(J) 190 \$))+(XC(1)*XC(J)+YC(1)*YC(J)-XC(1)*PYC(J)+YC(1)*PXC(J))*SIN(AL1(J)) 191 \$+ XC (J) * PXC (J) + YC (J) * PYC (J) 192 DD1=2.0*YC(1)*PXC(J)-2.0*XC(1)*PYC(J)-YC(1)*PYC(J)-XC(1)*PYC(J)+ 193 194 \$XC(1) + XC(J) + YC(1) + YC(J) DD2=2.0*XC(1)*PXC(J)+2.0*YC(1)*PYC(J)-XC(1)*PPYC(J)+YC(1)*PPXC(J)-195 **\$YC(1)***XC(J)+XC(1)*YC(J) 196 R(2)=COS(AL1(J))*DD1+SIN(AL1(J))*DD2+XC(J)*PPXC(J)+YC(J)*PPYC(J)+P 197 198 \$XC(J)*PXC(J)+PYC(J)*PYC(J) DDD5=-YC(1)*PPPYC(J)-XC(1)*PPPXC(J)+3.0*YC(1)*PPXC(J)-3.0*XC(1)*PP 199 \$YC(J)+3.0*XC(1)*PXC(J)+3.0*YC(1)*PYC(J)-YC(1)*XC(J)+XC(1)*YC(J)* 200 DDD 6=-XC(1)*PPP YC(J)+YC(1)*PPPXC(J)+3.0*XC(1)*PPXC(J)+3.0*YC(1)*PP 201 \$YC(J)-3.0*YC(1)*PXC(J)+3.0*XC(1)*PYC(J)-XC(1)*XC(J)-YC(1)*YC(J) 202 R(3)=COS(AL1(J))*DDD5+SIN(AL1(J))*DDD6+XC(J)*PPPXC(J)+YC(J)*PPYC(203 204 \$J)+3.0*PXC(J)*PPXC(J)+3.0*PYC(J)*PPYC(J) 205 DDDD5=-XC(1)*PPPPXC(J)-YC(1)*PPPPYC(J)+4.0*YC(1)*PPPXC(J)-4.0*XC(1 206 \$)*PPPYC(J)+6.0*XC(1)*PPXC(J)+6.0*YC(1)*PPYC(J)-4.0*YC(1)*PXC(J)+4. 207 \$0*XC(1)*PYC(J)-XC(1)*XC(J)-YC(1)*YC(J) DDDD6=-XC(1)*PPPYC(J)+YC(1)*PPPXC(J)+4.0*XC(1)*PPPXC(J)+4.0*YC(1 208 209 \$)*PPPYC(J)-6.0*YC(1)*PPXC(J)+6.0*XC(1)*PPYC(J)-4.0*XC(1)*PXC(J)-4. 210 \$0*YC(1)*PYC(J)+YC(1)*XC(J)-XC(1)*YC(J) R(4)=COS(AL1(J))*CDDD5+SIN(AL1(J))*DDDD6+XC(J)*PPPPXC(J)+YC(J)*PPP 211 \$PYC(J)+4.0*PXC(J)*PPPXC(J)+4.0*PYC(J)*PPPYC(J)+3.0*PPXC(J)*PPXC(J) 212 \$+3.0*PPYC(J)*PPYC(J) 213 P(1) = SIN(AL1(J))214 215 P(2)=COS(AL1(J)) 216 P(3) = -SIN(AL1(J))

CARD 217 P(4) = -COS(AL1(J))Q(1)=COS(AL1(J)) 218 Q(2) = -SIN(AL1(J))219 Q(3) = -COS(AL1(J))220 Q(4) = SIN(AL1(J))221 222 CALL SIMQ(H,R,4,KS) CALL SIMO(H1, P, 4, KS1) 223 CALL SIMO (H2,0,4,K52) 224 225 WRITE (6,99) KS 226 WRITE (6,99) KS1 227 WRITE (6.99) KS2 228 99 FORMAT (1X.11) 229 F1=Q(1)*Q(3)+Q(2)*Q(4)F2=P(1)*Q(3)+Q(1)*P(3)+P(2)*Q(4)+Q(2)*P(4)230 F3=R(1)*Q(3)+C(1)*R(3)+Q(2)*R(4)+Q(4)*R(2)231 F4=P(1)*P(3)+P(2)*P(4)232 F5=P(1)*R(3)+P(3)*R(1)+P(2)*R(4)+P(4)*R(2)-1.0 233 F6=R(1)*R(3)+R(2)*R(4) 234 235 $G_{1=0(1)*0(4)-0(2)*0(3)}$ $G_{2}=P(1)*Q(4)+Q(1)*P(4)-P(2)*Q(3)-Q(2)*P(3)$ 236 G3=R(1)+Q(4)+Q(1)+R(4)-R(2)+Q(3)-Q(2)+R(3)-1.0 237 G4=P(1)*P(4)-P(2)*P(3)238 G5=P(1)*R(4)+R(1)*P(4)-P(2)*R(3)-P(3)*R(2)239 $G_{6=R(1)*R(4)-R(2)*R(3)}$ 240 XCDF(1)=F1*F1*G6*G6+G3*G3*F1*F6-G3*G6*F1*F3+G1*G6*F3*F3+G1*G1*F6*F 241 \$6-G1*G3*F3*F6-2.0*G1*G6*F1*F6 242 $X(0 \in I_2) = 2 \cdot 0 \times 65 \times 66 \times 11 \times 11 \times 2 \cdot 0 \times 62 \times 63 \times 11 \times 165 \times 63 \times 63 \times 11 \times 155 - 63 \times 66 \times 11 \times 122 - 62$ 243 \$*G6 *F1 *F3-G3 *G5 *F1 *F3+2.0*G1 *G6 *F2 *F3+G1 *G5 *F3 *F3+2.0*G1 *F5 *F6-244 \$G1*G3*F2*F6-G1*G2*F3*F6-G1*G3*F3*F5-2.0*G1*G5*F1*F6-2.0*G1*G6*F1*F 245 246 \$5 XCOF(3)=2.0*G4*G6*F1*F1+G5*G5*F1*F1+G2*G2*F1*F6+2.0*G2*G3*F1*F5+G3 247 \$*G 3*F 1*F 4-G 2*G 6*F 1*F2-G3*G5*F1*F2-G2*G5*F1*F3-G3*G4*F1*F3+G1*G6*F2 248 \$*F2+2.0*G1*G5*F2*F3+G1*G4*F3*F3+2.0*G1*G1*F4*F6+G1*G1*F5*F5-G1*G2* 249 250 \$F2+F6-G1+G3+F2+F5-G1+G2+F3+F5-G1+G3+F3+F4-2.0+G1+G6+F1+F4-2.0+G1+G 251 \$5*F 1*F 5-2.C*G 1*G4*F 1*F6 252 XC0F(4)=2.0+G4+G5+F1+F1+G2+G2+F1+F5+2.0+G2+G3+F1+F4-G2+G5+F1+F2-G3 253 \$*G4*F1*F2-G2*G4*F1*F3+G1*G5*F2*F2+2.0*G1*G4*F2*F3+2.0*G1*G1*F4*F5-\$G1*G2*F2*F5~G1*G3*F2*F4~G1*G2*F3*F4~2.0*G1*G5*F1*F4~2.0*G1*G4*F1*F 254 255 \$5 XCDF(5)=G4*G4*F1*F1+G2*G2*F1*F4-G2*G4*F1*F2+G1*G4*F2*F2+G1*G1*F4*F 256 \$4-G1*G2*F2*F4+2.0*G1*G4*F1*F4 257 CALL POLRT (XCCF, COF, 4, ROOT R, ROOT I, IER) 258 WRITE (6,109) 259 109 FORMAT (/,27X,9F L1 ROOTS,/) 260 261 DC 110 I=1.4 262 110 WRITE (6,112) ROOTR(1),ROOTI(1) 112 FORMAT (17X, E13.5, 10X.E13.5,/) 263 WRITE (6,114) IER 264 114 FORMAT (1X, I1) 265 266 WRITE (6,106) 106 FORMAT (1H1,//) 267 125 DO 215 I=1,4 268 269 OA = F10B=F2*R00TR(1)+F3 270

	C00000001111111111222222222223333333333444444445555555555
CARD	15342018301534201830153420183015342018301534201830153420183015342018301534201830
271	$\Omega C = E 4 \times R C \Omega T R (I) \times R \Omega \Omega T R (I) + E 5 \times R \Omega \Omega T R (I) + E 5$
272	
273	$OE = 62 \times ROOTR(I) + 63$
274	DF=G4*R03TR(I)*R00TR(I)+G5*R00TR(I)+G6
275	IF (ABS(RODTI(1)),LT.0.0000001) RODTI(1)=0.0
276	IF (ROOTI(I)) 200,130,200
277	200 WRITE (6,201)
278	201 FORMAT (5X,18H RCOT IS IMAGINARY,/)
279	GO TO 215
280	130 SA=0B*0B-4.0*0A*0C
281	IF (SA) 200,140,140
282	140 ROOTA 1= (-OB+ SQR T(SA)) /(2+0*0A)
283	ROOT A2= (-OB-SQRT (SA))/(2.0*OA)
284	SB=OE+OE-4.0*CD*CF
285	IF (S8) 200,150,150
286	150 ROOTB1=(-OE+SQRT(SB))/(2+0*00)
287	R00TB2=(-OE-SQRT(SB))/(2.0*OD)
288	WRITE (6,155) ROOTA1,ROOTA2,ROOTB1,ROOTB2
289	155 FORMAT (/,13X,8+ ROOTA1=,F18.6,5X,8H ROOTA2=,F18.6,/,13X,8H ROOTB1
290	\$=,F18.6,5X,8H RCCTB2=,F18.6,/)
291	IF (ABS(ROOTA1-ROOTB1)+LT.0.05) GO TO 160
292	IF (ABS(RCOTA1-ROOTB2).LT.0.05) GO TO 160
293	IF (ABS(ROOTA2-ROOTB1).LT.0.05) GO TO 170
294	IF (ABS(ROOTA2-ROOTB2).LT.0.05) GO TO 170
295	GO TO 215
296	160 WRITE (6,180) ROOTA1
297	180 FORMAT (2X,16H SECCND ROOT IS ,F20.5,/)
298	R OD T=ROGTA 1
299	GO TO 210
300	170 WRITE (6,180) RCCTA2
301	ROOT=ROOTA2
302	210 WRITE (6,212)
303	212 FORMAT (14X,13H CENTER POINT,31X,13H CIRCLE POINT,/,8X,3H XA,19X,3
304	\$H YA,19X,3H XB,19X,3H YB,/)
305	XA(1)=R(1)+P(1)*R00TR(I)+Q(1)*R00T
306	YA(I)=R(2)+P(2)+RODTR(I)+Q(2)+RODT
307	XB(I)=R(3)+P(3)+ROOTR(I)+Q(3)+ROOT
308	YB(1)=R(4)+P(4)+R00TR(1)+Q(4)+R00T
309	WRITE (6,230) XA(1), YA(1), XB(1)
310	230 FORMAT (4F20.8,/,33X,20H XXXXXXXXXXXXXXXXXX//)
311	215 CONTINUE
312	STOP
313	END
312 313	STOP END

VITA "

William Richard Coutant, Jr.

Candidate for the Degree of

Master of Science

Thesis: SYNTHESIS OF STEPHENSON TYPE II SIX-LINK FUNCTION GENERATOR FOR FINITELY AND INFINITESIMALLY SEPARATED POSITIONS

Major Field: Mechanical Engineering

Biographical:

- Personal Data: Born in Manhattan, Kansas, July 31, 1950, the son of Dr. and Mrs. W. R. Coutant.
- Education: Graduated from Sooner High School, Bartlesville, Oklahoma, in May, 1968; attended the University of Kansas at Lawrence, 1968-1969; received the Bachelor of Science degree in Mechanical Engineering from Oklahoma State University in December, 1972; completed requirements for the Master of Science degree at Oklahoma State University in December, 1973; member of Pi Tau Sigma and the American Society of Mechanical Engineers.
- Professional Experience: Engineer Aide, United States Bureau of Mines, 1971; graduate research assistant, School of Mechanical Engineering, Oklahoma State University, 1973.