

TRUSS-ANALOGY METHOD FOR THE ANALYSIS
OF STEEL DIAPHRAGM

By

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Diploma

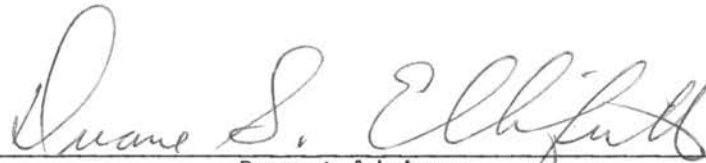
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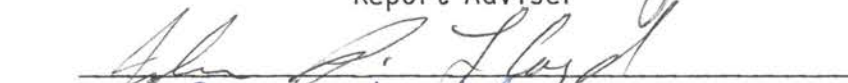
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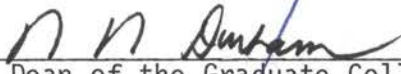
TRUSS-ANALOGY METHOD FOR THE ANALYSIS
OF STEEL DIAPHRAGM

Report Approved:



Report Adviser



Dean of the Graduate College

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TABLE OF CONTENTS

Chapter	Page
I. INTRODUCTION	1
1.1 General	1
1.2 Brief Description of West Virginia Tests	2
1.3 Scope and Objective of Investigation	2
II. APPROACH OF TRUSS-ANALOGY METHOD	7
III. DESCRIPTION OF THE TRUSS-ANALOGY METHOD	9
IV. CORRELATION WITH TEST RESULTS	12
V. MATHEMATICAL MODELS	21
5.1 General	21
5.2 Limitations of the Mathematical Models	21
VI. SUMMARY AND CONCLUSIONS	27
VII. RECOMMENDATIONS FOR FUTURE RESEARCH	29
REFERENCES	31
APPENDIX - "STRUDL" COMPUTER INPUT LIST	32

LIST OF FIGURES

Figure	Page
1. Diaphragm Test Frame	3
2. Welding Steel Deck to Test Frame	4
3. Diaphragm Loading Device	5
4. Bryan Test Specimen	8
5. Relationship Between Diagonal Stresses and Shear Load	8
6. Illustration Example	11
7. Through 15. Correlation Curves]3 -	20
16. Cross-Section Area V.S. L/T Ratio	22
17. Buckling Load V.S. L/T Ratio	23
18. Effectiveness of Steel Deck Illustration	30
19. Typical Configuration of Truss Analogy	32

CHAPTER I

INTRODUCTION

1.1 General

The behavior of light gage steel panel diaphragms does not yield nicely to analysis. The large number of relatively small parts involved, with possible individual movement, and the stress concentrations that are present near the welds as local buckling and tearing around welds when horizontal load is applied, prevent the application of conventional methods of analysis with large degree of confidence. Accordingly, a considerable number of isolated tests have been performed by a number of persons and institutions over a period of time.

Among these tests, the most recent were conducted by the Department of Civil Engineering of West Virginia University in the late 1960's. There followed an extended series of more than one hundred diaphragm tests, utilizing many types of panels, steel thickness, patterns of welds, panel spans, and panel depths. Information that has evolved from these tests has provided a firm basis for the design and installation of light gage steel diaphragms in many parts of the country. Ultimate strength and working strength values have been established for many different systems, and the performance of such diaphragms under load has been accurately cataloged.

All the mathematical models developed in this report are adjusted according to these test results.

1.2 Brief Description of West Virginia Tests

At West Virginia University a large-scale diaphragms testing program was begun in 1967. Tests were made on 16, 18, 20, and 22 gage decks with lengths of 12, 16 and 20 feet long. Panel widths tested were 18, 24, 30, and 36 inches. All tests were made on a horizontal cantilever test frame, illustrated in Figure 1. Connections between the perimeter members of the frame, as well as connections at the purlin ends, were made with light clip angles and bolts. The entire frame assembly was supported on a roller system and could be moved easily prior to attaching the deck, indicating that all interior connections could be considered pinned. The steel deck was then welded to the frame, thus creating a shear-rigid diaphragm (see Figure 2). The diaphragm was then loaded in its plane by a hydraulic jack and load cell arrangement in line with the free edge (Figure 3). Load was applied in increments from zero to failure with deflection measurements made at each stage of loading. For more detail arrangement and test procedures, see Reference 1.

1.3 Scope and Objective of Investigation

Tested diaphragms were evaluated with respect to two major behavioral parameters, ultimate strength and shear stiffness. From the West Virginia University research, it is apparent that strength and stiffness are primarily influenced by sheet thickness, purlin spacing, panel width, panel length, yield strength of material, deck shape, fastener type, and arrangement. The purpose of this report is to develop a truss-analogy method to determine these two parameters in

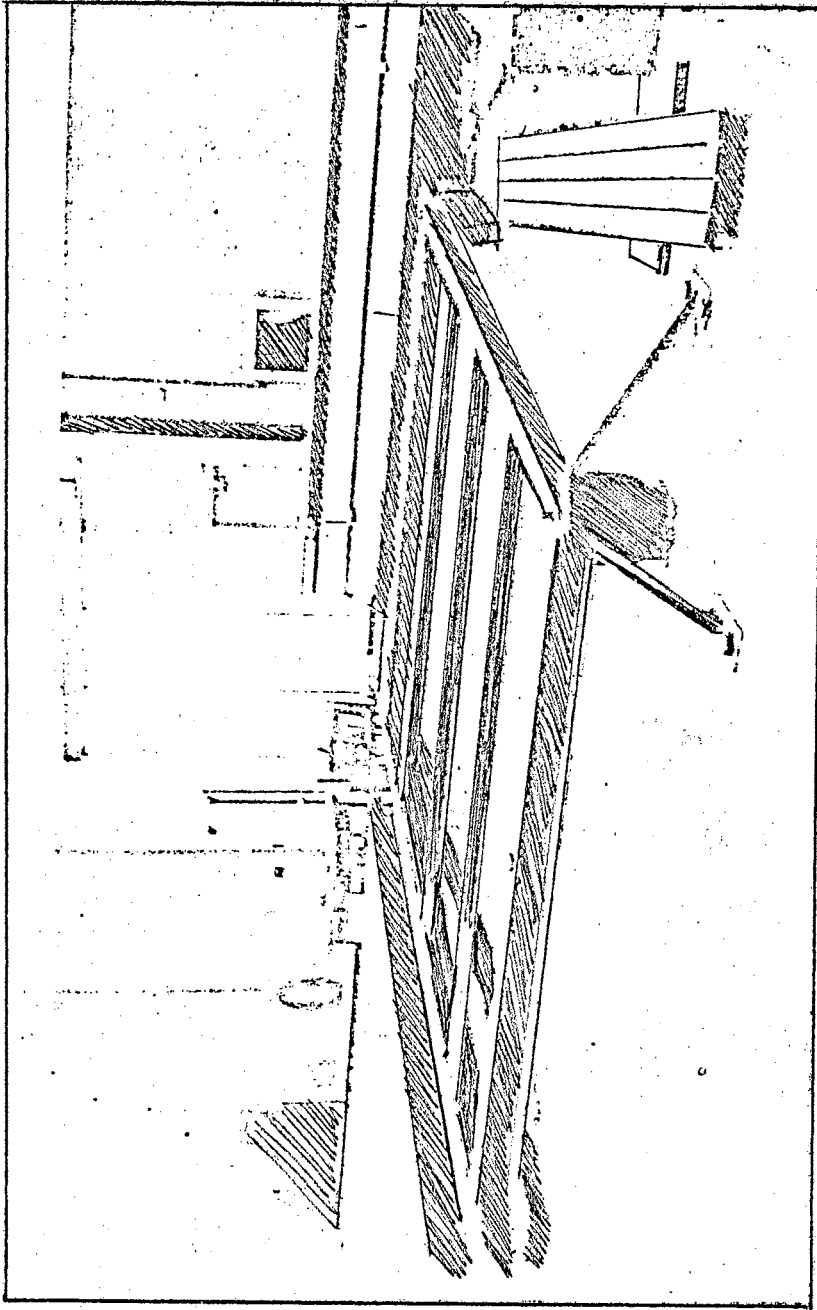


Figure 1. Diaphragm Test Frame

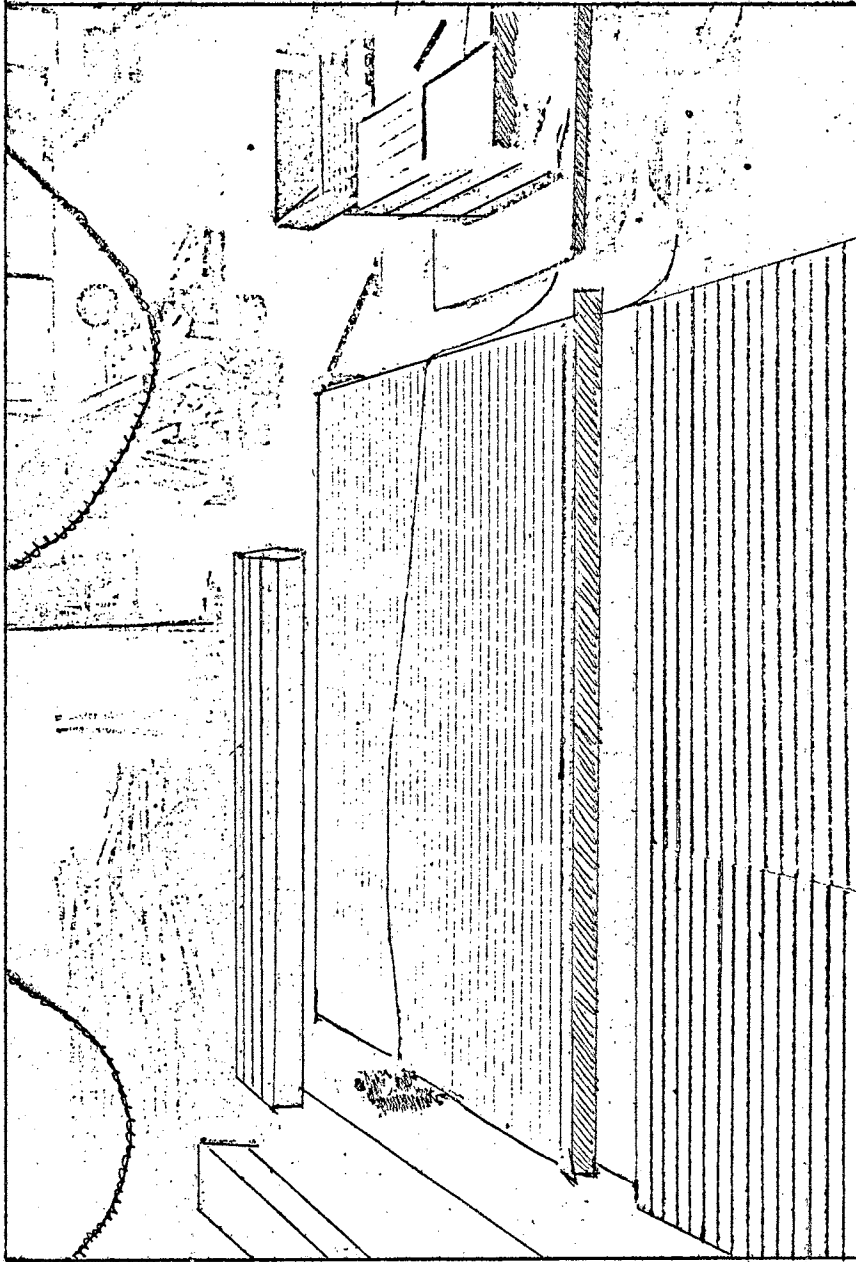


Figure 2. Welding Steel Deck to Test Frame

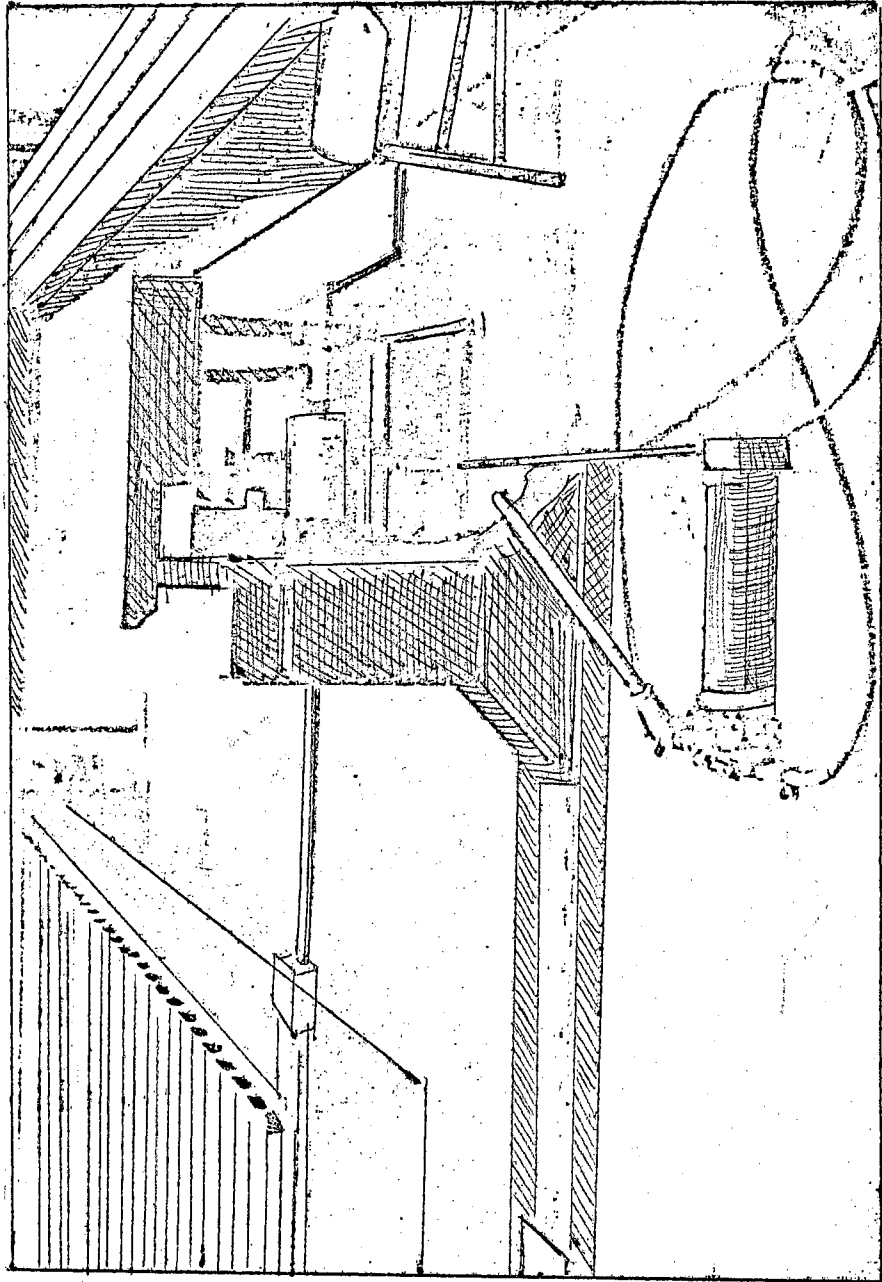


Figure 3. Diaphragm Loading Device

terms of the most significant of the variables mentioned above. The report is divided into sections covering the approach of the truss-analogy method, description of the method, development of empirical equations, correlations with test data, and conclusions. An example problem is included in the Appendix.

CHAPTER II

APPROACH OF TRUSS-ANALOGY METHOD

In 1964, Eric R. Bryan conducted a series of tests on shear of thin plates. The apparatus consisted of an aluminum sheet, 18 inches by 12 inches by 0.01 inches thick. The edge members, also of aluminum, are pinned at the corners (as shown in Figure 4) so that all the shear is carried by the sheet. There are four straingages attached in tensile diagonal and compressive diagonal directions. A plot of test results which indicates the relationship between shear load Q and diagonal stresses is shown in Figure 5.

It can be seen that, after buckling, the compressive stresses do increase because of the restraint of tensile field, but at a much lower rate than the tensile stresses. They also become asymptotic to a certain value, whereas the tensile stresses continue to increase. As a result, the compressive stress, after buckling, is a small percentage of the tensile stress as the external load is increased. Consequently, the excessive compressive stress then is carried by its panel edge members. If the panel edge member is a part of the steel deck itself, which will be discussed later, the buckling failure will be the panel edge flute buckling failure.

Based on the above analysis, the truss-analogy approach, with compressive diagonal members ignored, is constructed.

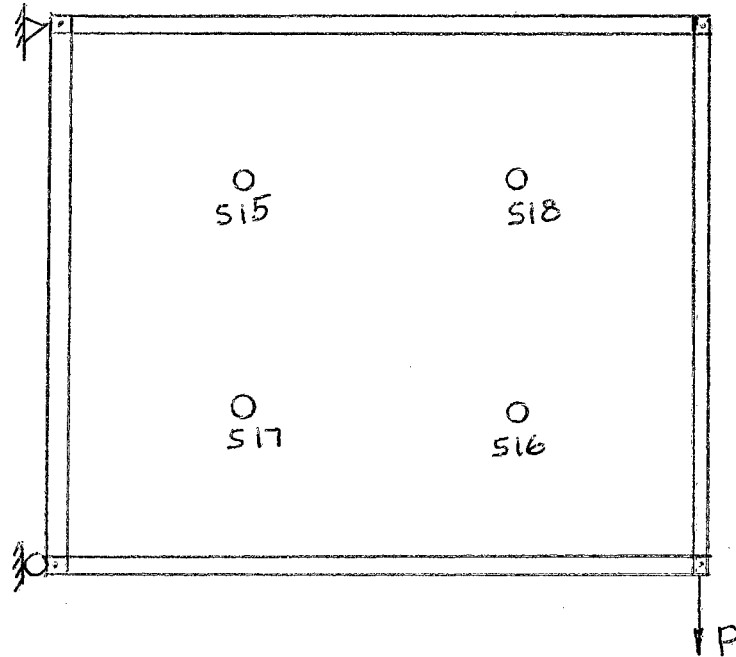


Figure 4. Bryan Test Specimen

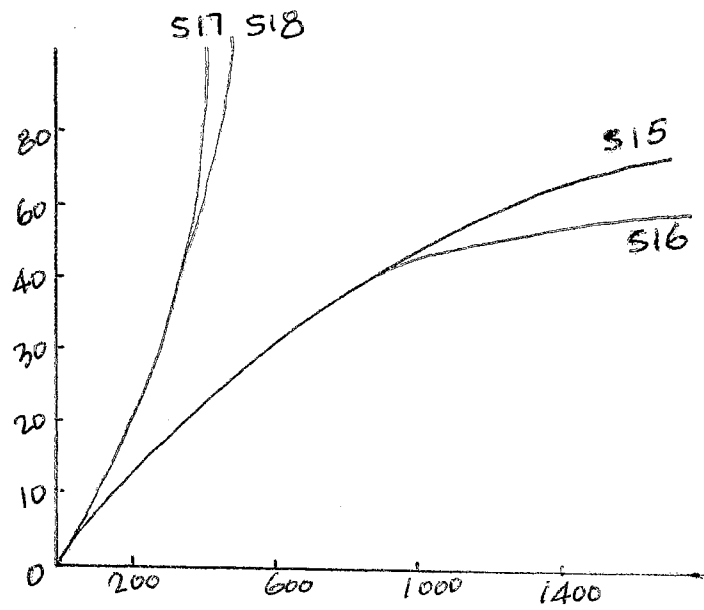


Figure 5. Relationship Between Diagonal Stresses and Shear Load

CHAPTER III

DESCRIPTION OF THE TRUSS-ANALOGY METHOD

Since the truss-analogy structure is a highly statically indeterminate structure, it is desirable to use a computer to solve. As far as I know, there are two computer programs available; these are "Plane Frame and Truss Program" and "STRUDL" program. The former limits the number of truss members to not more than 400 and joints to not more than 200. In this report "STRUDL" is used. The following steps are used in the analytical procedures.

1. Set up a "STRUDL" coordinates system as shown in Figures 6 and 7.
2. All the member property input data are known except the cross-section of those imaginary internal truss members. A value for that was first assumed and this value could easily be adjusted later, because in the elastic medium the deflection of a member is directly proportional to its sectional area. Equation (1) shows the relationship between cross-sectional area, a , and deflection:

$$\frac{\Delta_t}{A_t} = \frac{\Delta_c}{A_c} \quad (1)$$

where Δ_t is the test data deflection, A_c is the assumed computer input data, A_t is the cross-sectional area needed, and Δ_c is the computer output result of deflection corresponding to A_c . Figure 17 is the plot of cross section, V.S. L/T ratio.

3. After the cross-sectional area of the diagonal has been determined, assume values of the buckling force and ultimate tensile force in the diagonal, then compare computer deflection results with test results. If the test results do not agree, readjust the previous assumptions. Continue this trial-and-error process until the computer results fit the test results in an acceptable region. Figure 18 is the plot of buckling force in compressive member V.S. L/T ratio.

4. As the trial-and-error process continued, it was found that after the steel deck was torn off around the welds, namely when the ultimate tensile force had been reached, it still resisted some force. This phenomenon can be explained: when one steel deck panel was torn off, the adjacent welds picked up the load that had formerly been carried by the weld that failed. Therefore, when any member reached its ultimate tensile force, it was taken out and fifty percent of its ultimate tensile force was applied to that joint in the direction of that member in the next computer run. On the other hand, when a member buckled, it was merely taken out (see Figure 8).

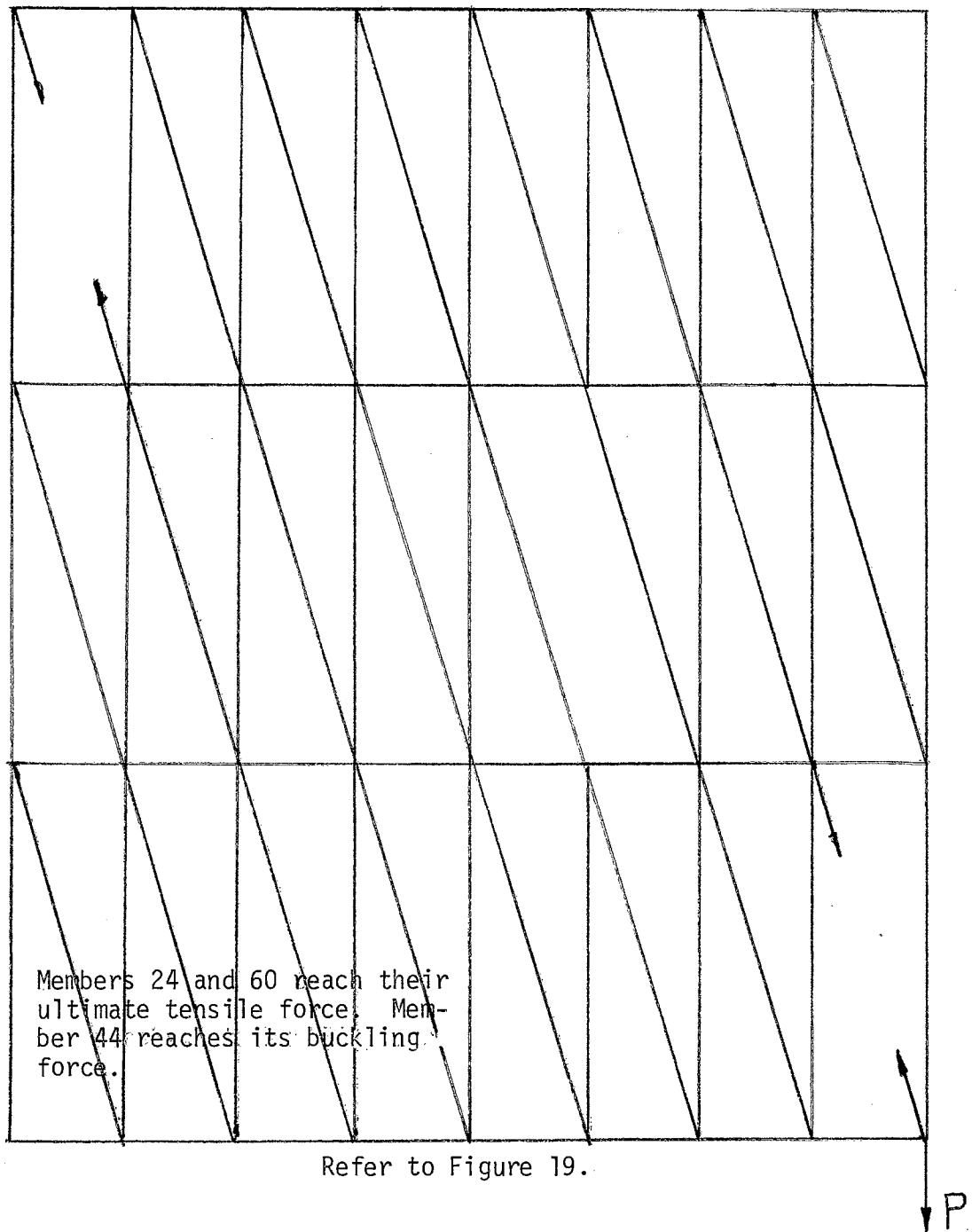


Figure 6. Illustration Example

CHAPTER IV

CORRELATION WITH TEST RESULTS

The following are ten correlation plots. From these plots, it is seen that the computer input assumptions are well confirmed.

The designations on the test curves, such as W-3, refer to tests made at West Virginia University.

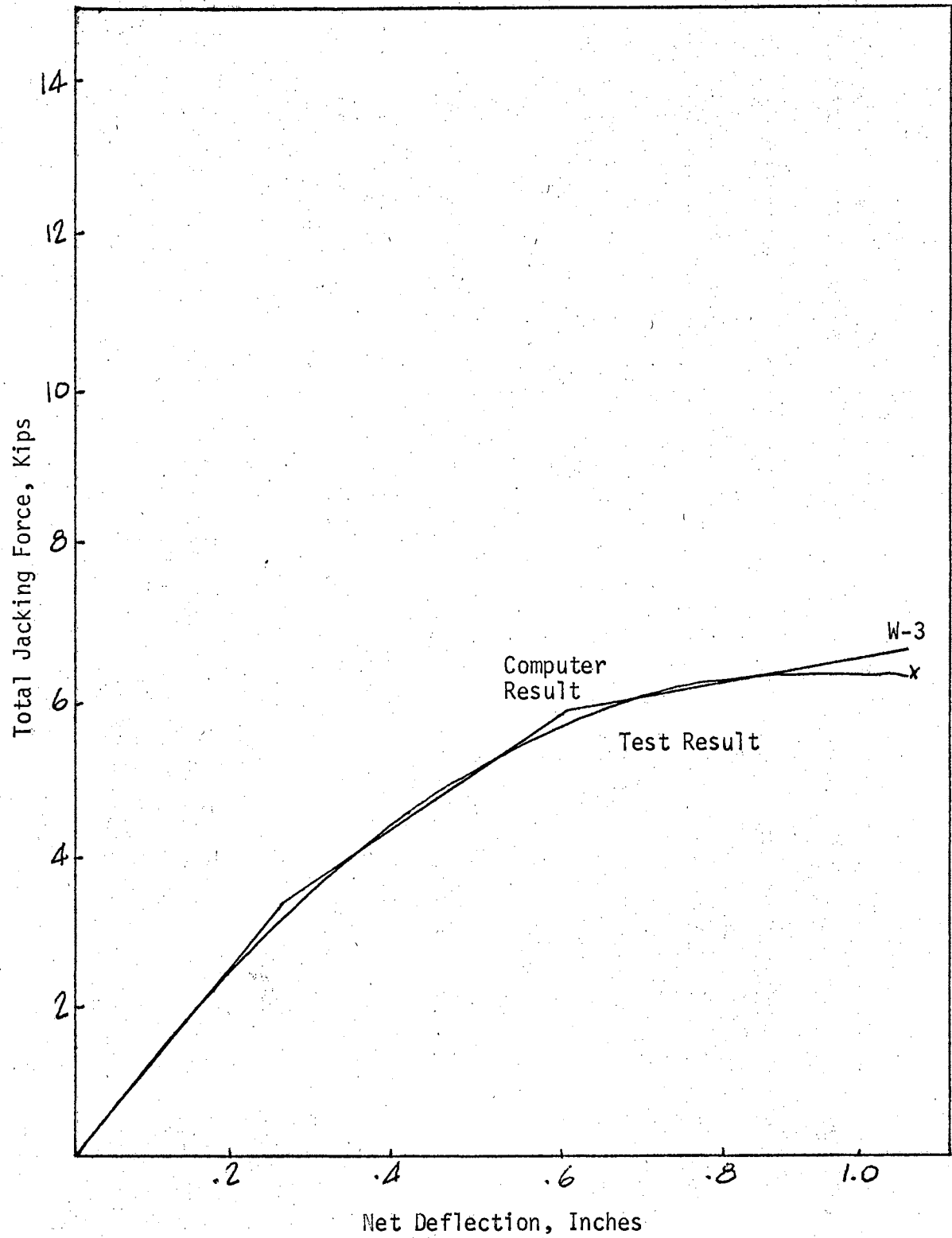


Figure 8. Correlation Curves

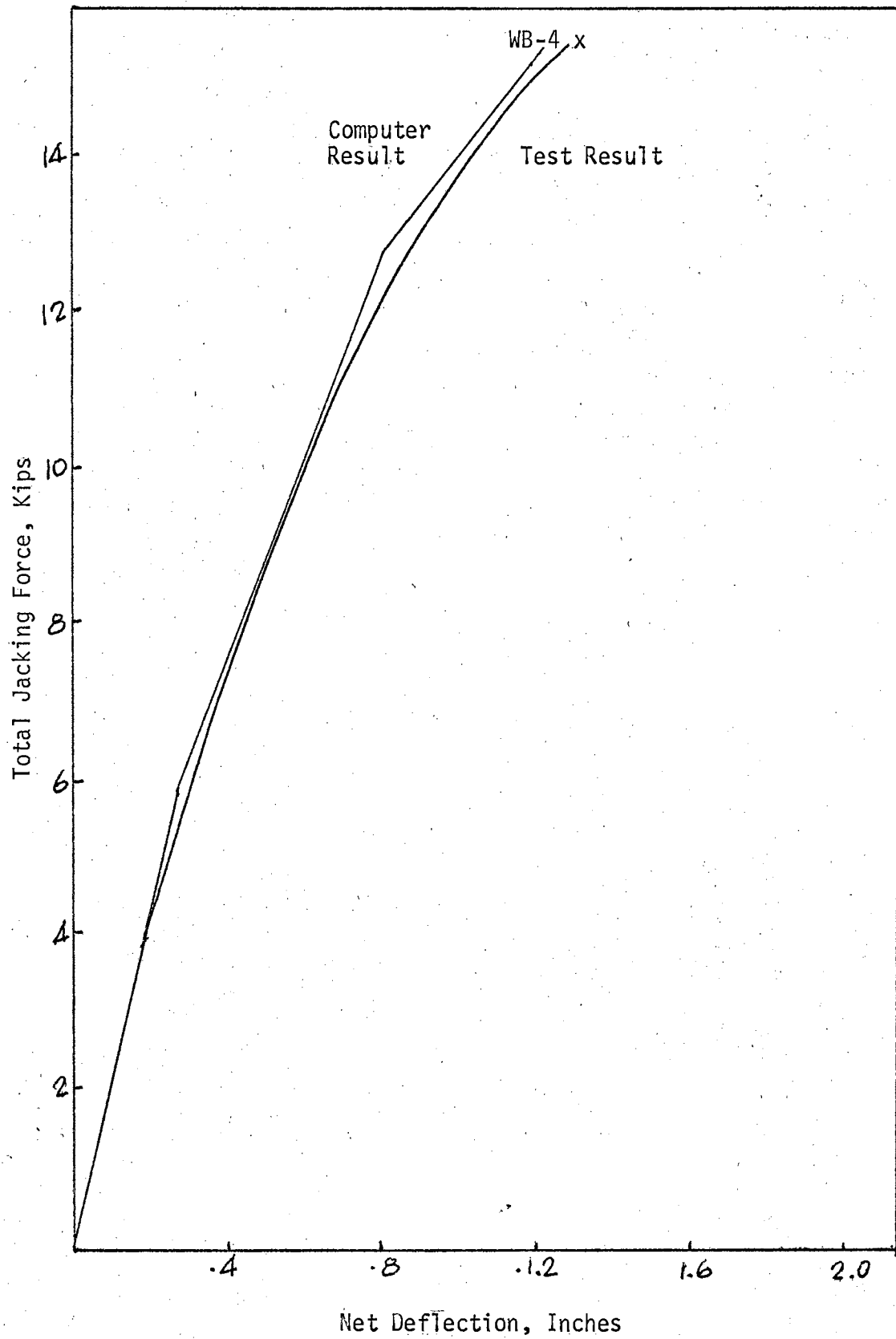


Figure 9. Correlation Curves

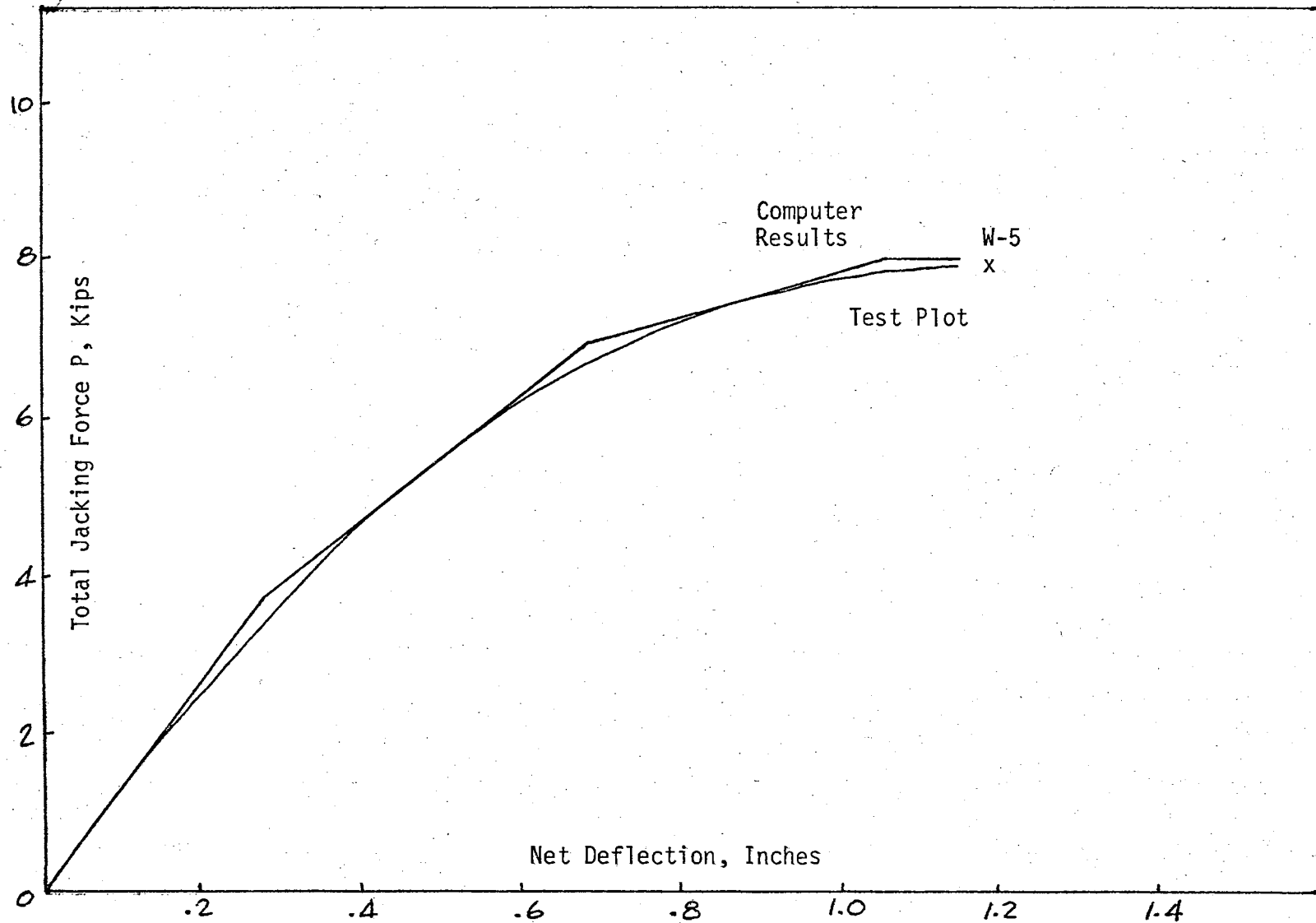


Figure 10. Correlation Curves

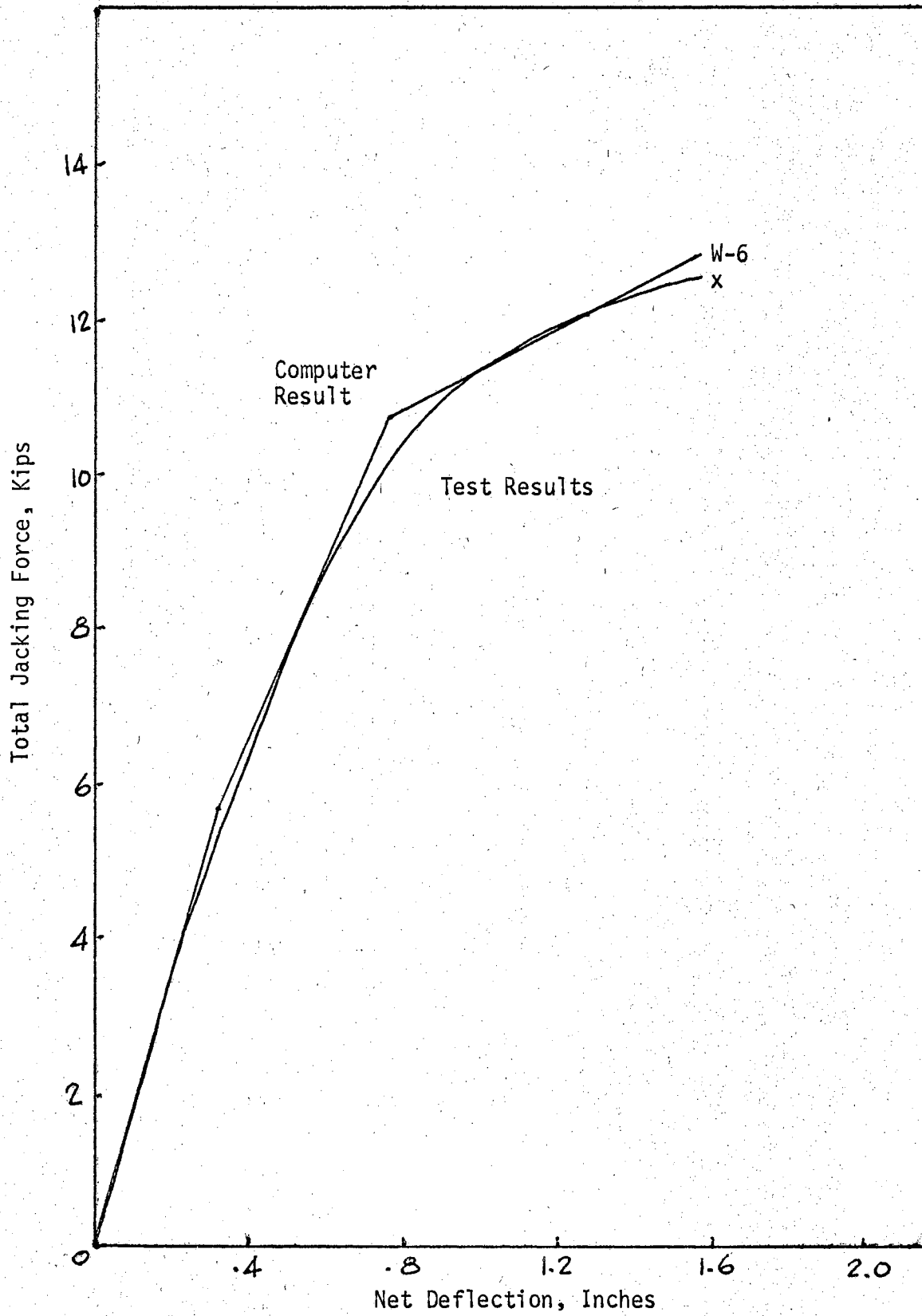


Figure 11. Correlation Curves

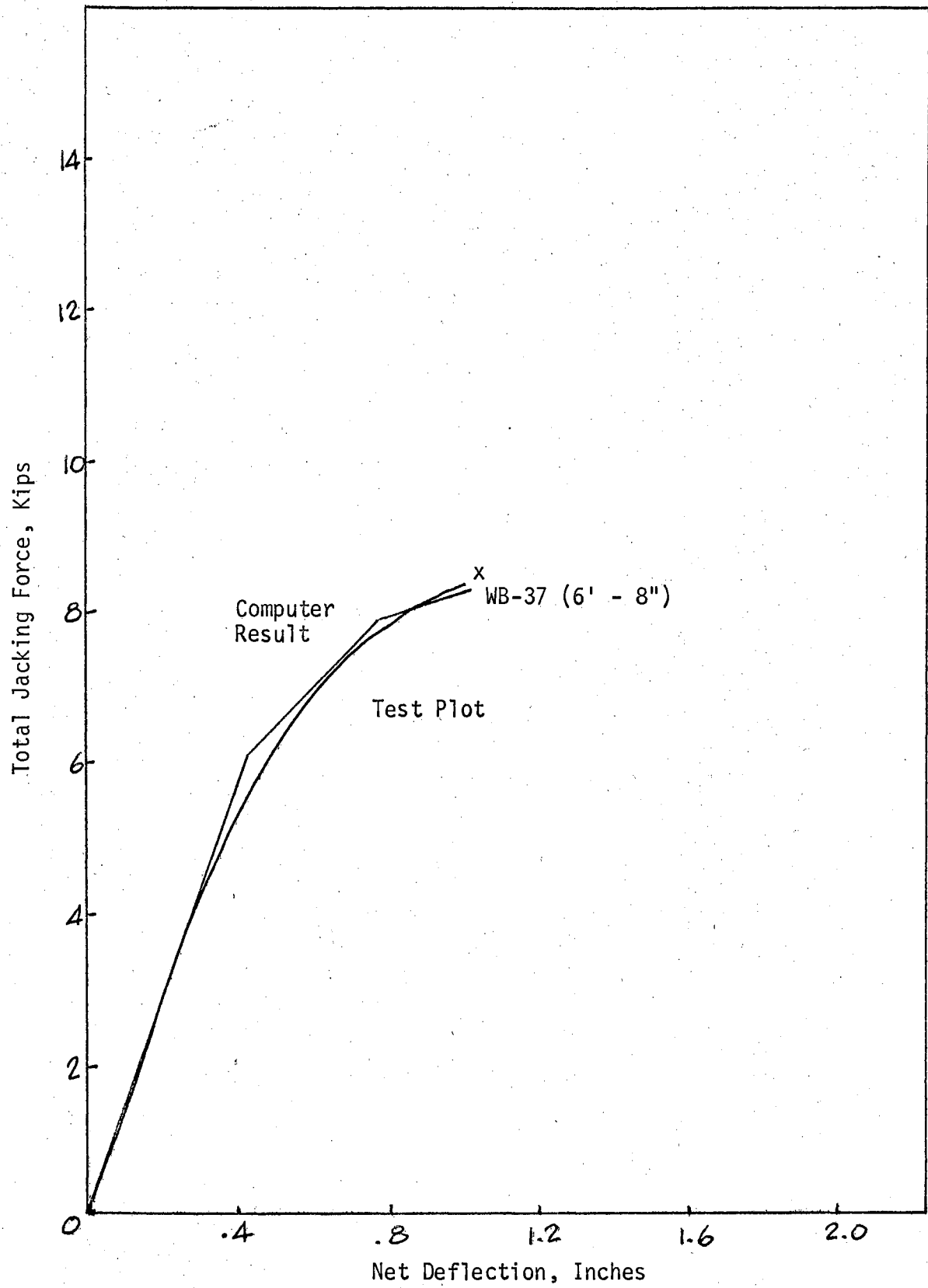


Figure 12. Correlation Curves

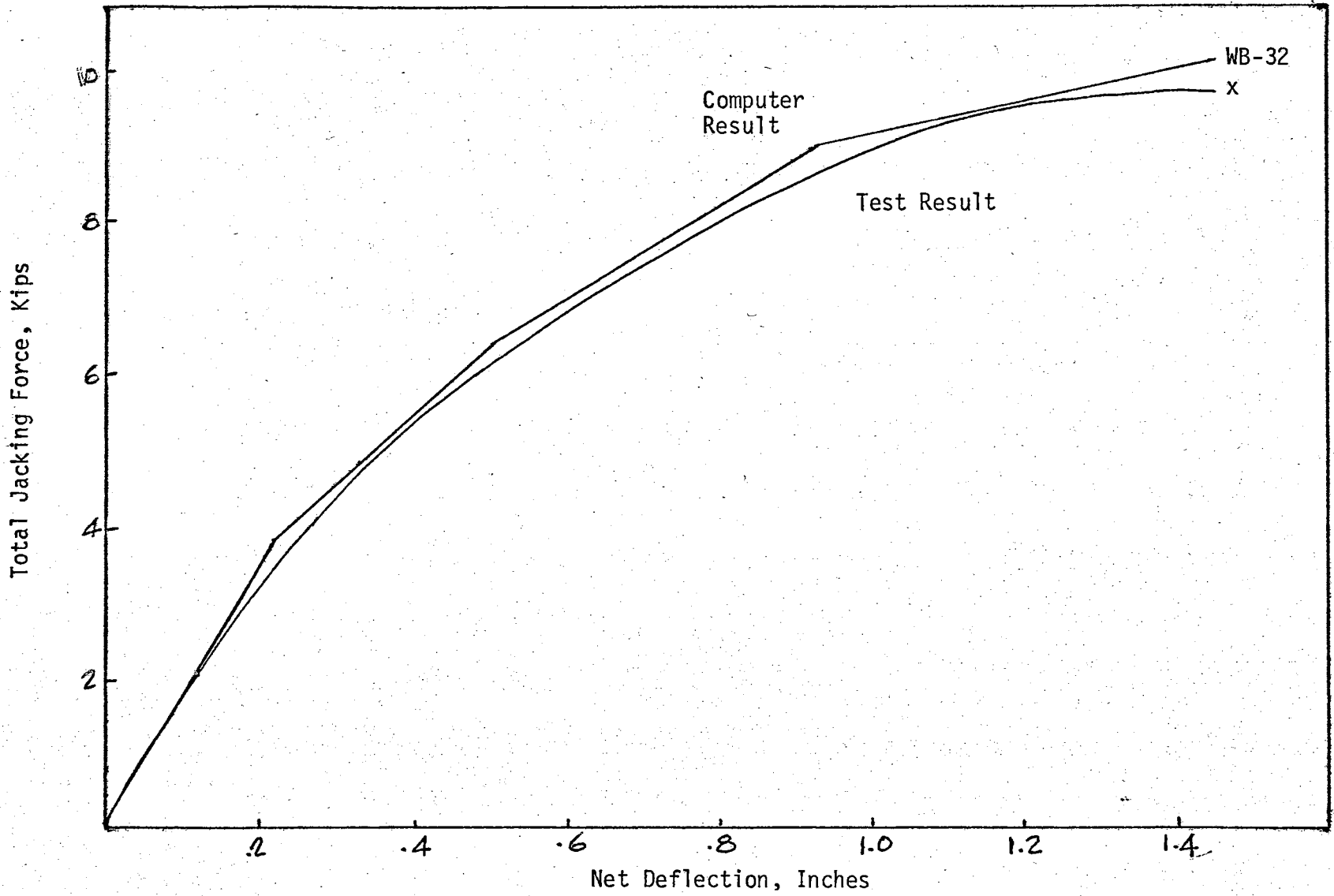


Figure 13. Correlation Curves

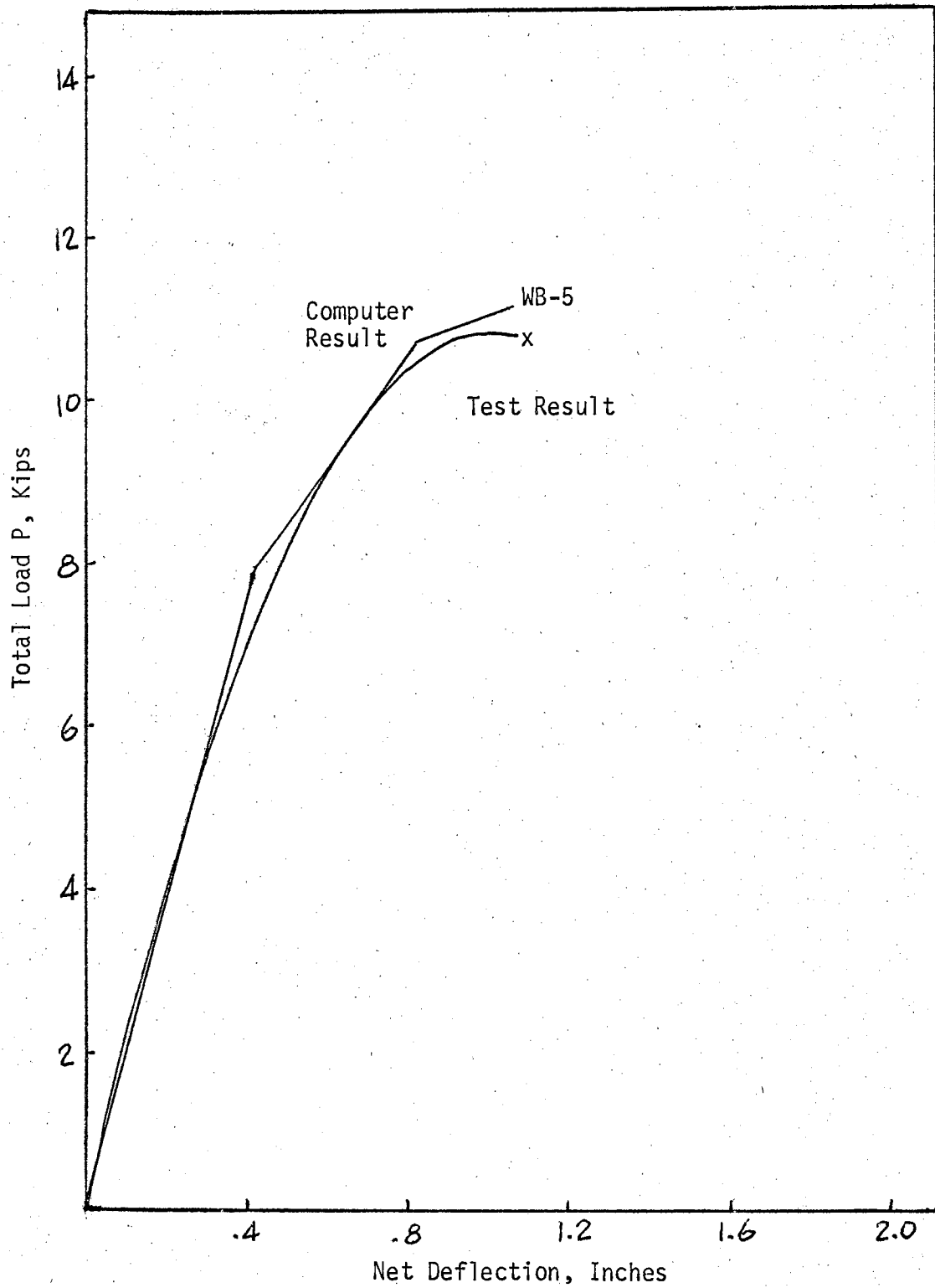


Figure 14. Correlation Curves

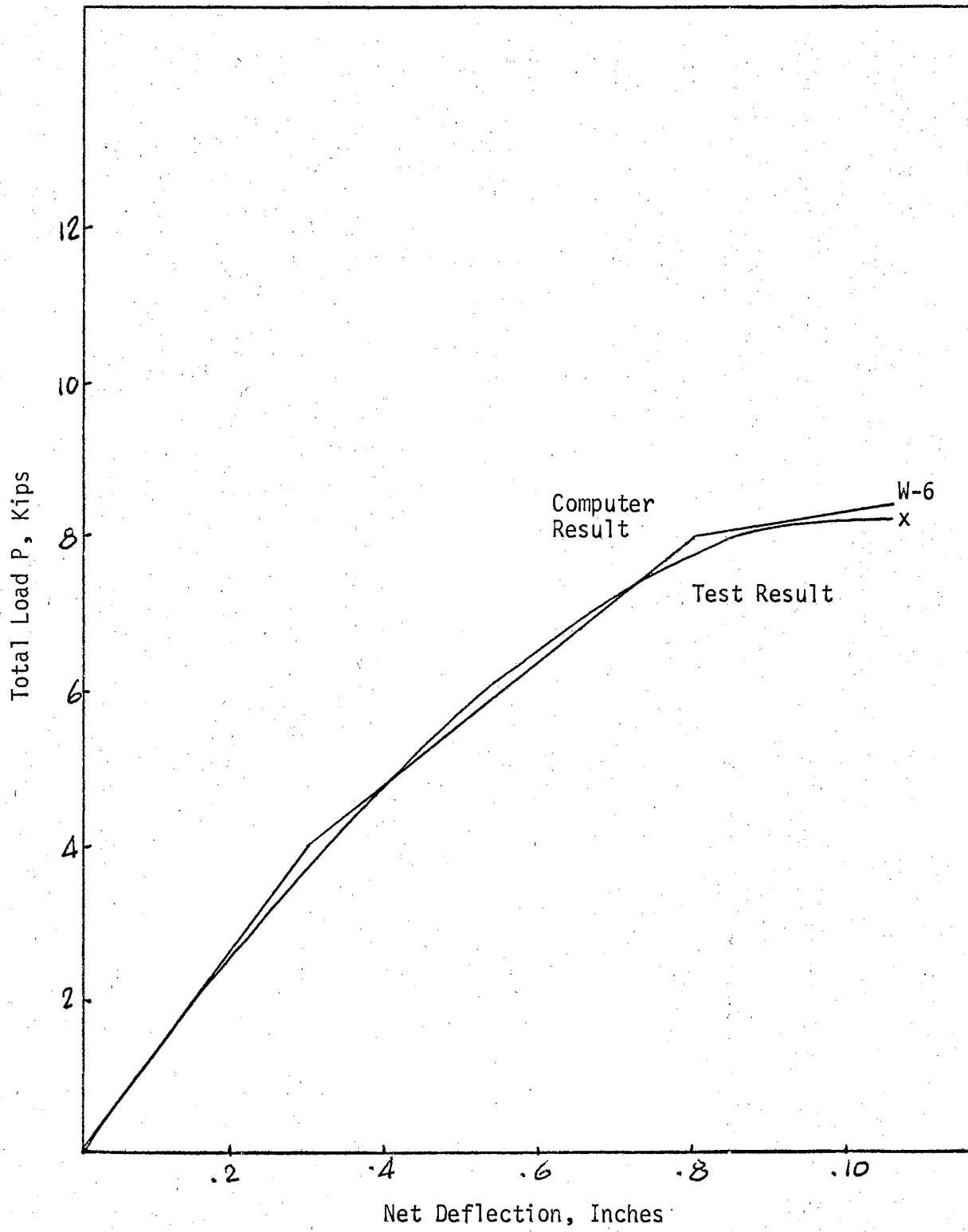


Figure 15. Correlation Curves

CHAPTER V

MATHEMATICAL MODELS

5.1 General

Since the length of purlin spacing, L , and the thickness of steel decks are the dominating parameters of shear strength of steel diaphragms, so the values of the section area, buckling force and ultimate tensile force equations herein are all expressed in terms of L/t ratio. All data curves are so plotted by using second order parabolic interpolation.

5.2 Limitations of the Mathematical Models

The mathematical models are:

1. Valid for W and WB type decks.
2. Used for standard weld patterns only.

Since the L/t ratio of the test steel diaphragms were ranged from 1000 to 3000, the mathematical models obtained from Figure 18 will be considerably accurate when L/t ratio within that range. The following equations are to the steel deck type specified as 22, 20 and 18 gage deck.

For 22 gage deck: (From $L/t = 1300$ to $L/t = 3000$)

$$a = 1.732 \times 10^{-7} L^2 t^{-2} + 89.7 \times 10^{-5} L t^{-1} - 0.974.$$

For 20 gage deck: (From $L/t = 1000$ to $L/t = 3000$)

$$a = -1.665 \times 10^{-7} L^2 t^{-2} + 68.57 \times 10^{-5} L t^{-1} - 0.478.$$

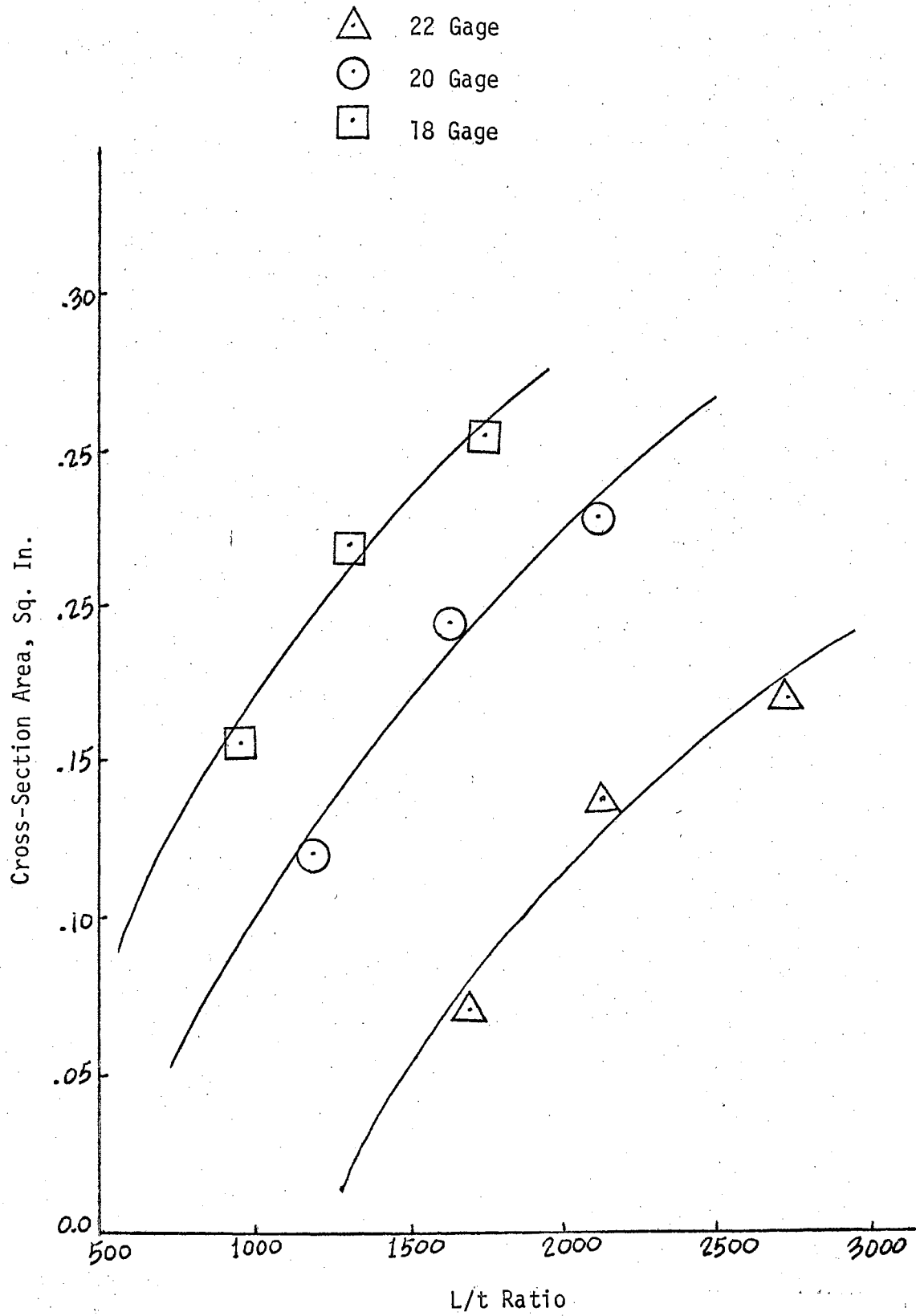


Figure 16. Cross-Section Area Versus L/t Ratio

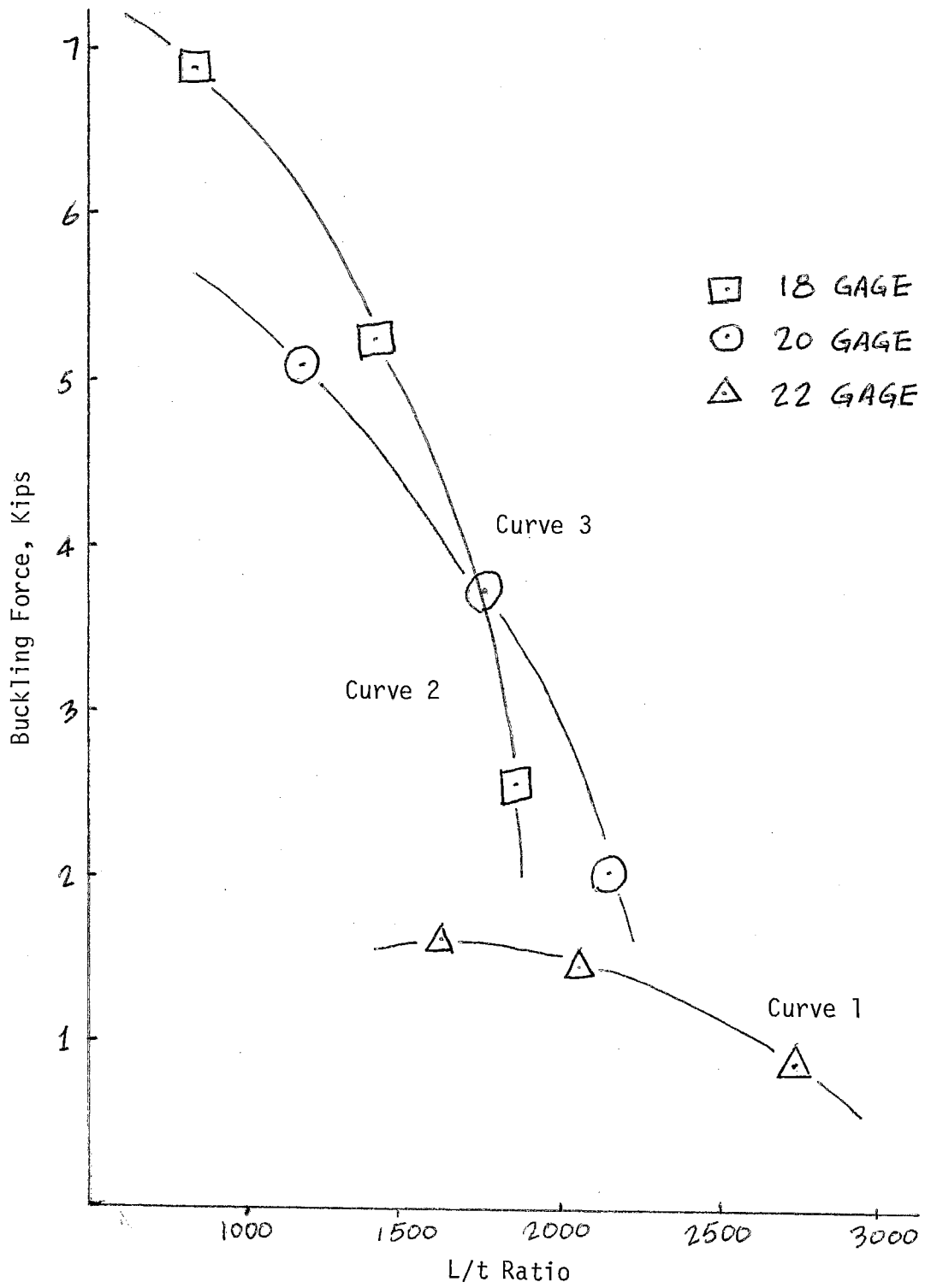


Figure 17. Buckling Loads Versus L/t Ratio

For 18 gage deck: (From $L/t = 500$ to $L/t = 2500$)

$$a = -2.74 \times 10^{-7} L^2 t^{-2} + 87.64 \times 10^{-5} L t^{-1} - 0.442.$$

Apparently, the member buckling force plotted in Figure 17 is the combination of the effects of local buckling and overall buckling. Since the steel deck is formed by a piece of thin steel plate, the thickness of the deck strongly influences the magnitude of local buckling load prior to overall buckling.

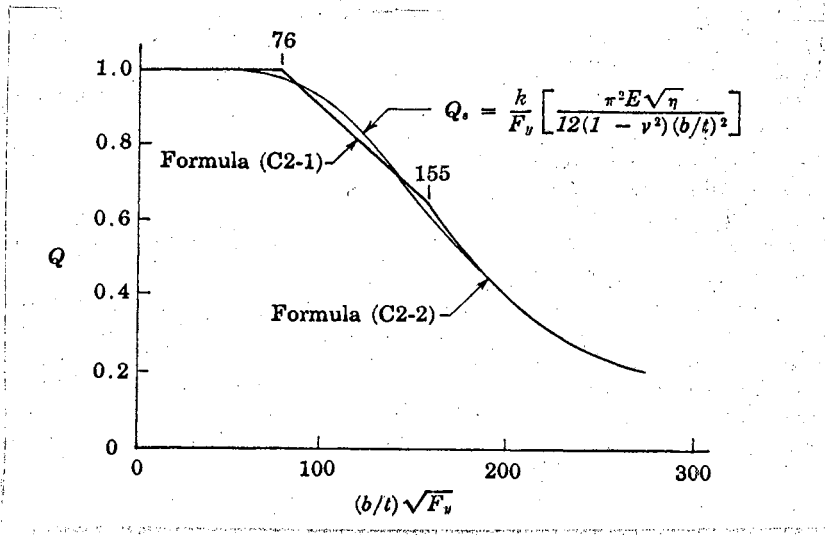
As it was pointed out in the introduction of this report, the behavior of light gage panel diaphragms does not yield nicely to analysis because the large number of relatively small parts involved. One of the most important properties among those small parts is the quality of welding which was assumed perfect in this method. Consequently, the buckling forces were obtained on the basis of this assumption.

Even though the quality of welding was inspected in the laboratory, it is necessary to investigate how good that assumption is, as far as the determination of buckling forces is concerned.

AISC presents an interaction diagram of axial force Q versus $(b/t) \sqrt{f_y}$ shown below. It indicates that when $b/t \sqrt{f_y} \geq 200$, the capacity of resisting axial load decreases to about 20 percent of its capacity when $b/t \sqrt{f_y} \leq 100$, and remains a low limit up to $b/t \sqrt{f_y} = 300$. In our case, assume $b/t \sqrt{f_y} = 330$ for 22 gage deck. Curve 1 represents 22 gage which is rather flat regardless of length L , so this conforms with the AISC curve.

For curve 2, $b/t \sqrt{f_y} = 180$, which represents 20 gage deck which is controlled by both $(b/t) \sqrt{f_y}$ ratio and Euler's $1/L^2$ ratio. Since $(b/t) \sqrt{f_y}$ ratio of 18 gage and 20 gage are both in the range of 100 to

200, from AISC's curve, the slope in this range is almost constant as is that in Figure 17.



The lowest point of each curve represents when $L = 6.8$ feet, the second lower point represents $L = 5'$, and the highest point represents $L = 4'$. From curves 2 and 3, it can be seen that the buckling force almost doubles as L decreases from $4'$ to $5'$ or from $5'$ to $6.8'$, which is consistent with the AISC column strength curve.

The following are the equations for 22, 20 and 18 gage deck separately:

22 Gage: (L/t ratio from 1500 to 2750)

$$F_a = -0.432 \cdot 10^{-6} L^2 t^{-2} + 13.79 \cdot 10^{-4} L t^{-1} + 0.501.$$

20 Gage: (L/t ratio from 1000 to 2500)

$$F_a = 1.178 \cdot 10^{-6} L^2 t^{-2} - 77.18 \cdot 10^{-4} L t^{-1} + 13.29.$$

18 Gage: (L/t ratio from 800 to 2000)

$$F_a = 3.474 \cdot 10^{-6} L^2 t^{-2} - 159 \cdot 10^{-4} L t^{-1} + 19.51.$$

Ultimate tensile force of truss member:

$$T = 3.50 t / .036.$$

CHAPTER VI

SUMMARY AND CONCLUSIONS

When the analytical results for the truss-analogy model were compared with the test data, four conclusions were reached:

1. The truss-analogy method provides some important information; that is, the stresses in steel decks are due to external in-plane load. From this additional information a designer can visualize what portions of a diaphragm are critical, and can predict how the stress of one deck panel is transmitted to adjacent panels after that panel fails.

2. This method is a valuable and rather unique way to solve a steel deck floor or roof with openings by removing members in absent panel areas during analysis.

3. It can be a tool for a designer to anticipate the capacity of a steel diaphragm which has already been built and contains some defective welds due to imperfect workmanship.

4. The mathematical expressions for calculating buckling load and ultimate tensile load have been developed in this method. After these two values of a particular design case have been obtained, it is justified and desirable for a designer to decide whether the extra welds are needed to strengthen the diaphragm by comparing the buckling force and ultimate tensile force. In other words, it would be senseless to do so if the buckling force controls.

Throughout the above four conclusions, it can be seen that the advantage of this method is to provide knowledge of how steel diaphragms behave under in-plane force and how to attack some design cases with configurations other than a rectangular shape.

CHAPTER VII

RECOMMENDATIONS FOR FUTURE RESEARCH

The following recommendations are offered for future research:

1. In many structures, shear-resistant light gage metal diaphragms are connected directly to beams or columns of the steel framework, and may continuously brace these members along their length. The light-gage wall cladding on a building frame, for example, can brace the columns against weak axis buckling if adequate connection is provided between the columns and the diaphragm. Similarly, light-gage steel roof or floor decking can restrain lateral buckling of truss chords, beams and purlins. This action of the diaphragm in bracing individual members has been investigated by Cornell University in 1967. However, if the truss-analogy method can be developed for buckling-restraint diaphragms, the advantages are obvious as indicated in the previous conclusions.

2. As in practice, the edge members of a light-gage diaphragm can be connected in several fashions, such as two corners pinned with the other two corners rigid or even semi-rigid. Suppose there was a four-corner, rigid-connected diaphragm, as shown in Figure 18 (the dotted line indicates the deflection curve under load P). As you can see, the steel deck in Zones 1 and 3 experience little shear strain. In other words, only the steel deck in Zone 2 was resistant to shear force. Therefore, what proportion of the shear capacity, after all corners

pinned shear diaphragm have been taken into account, is in question. After this question has been answered, the truss-analogy method can then be applied by removing members in Zone 1 and 3.

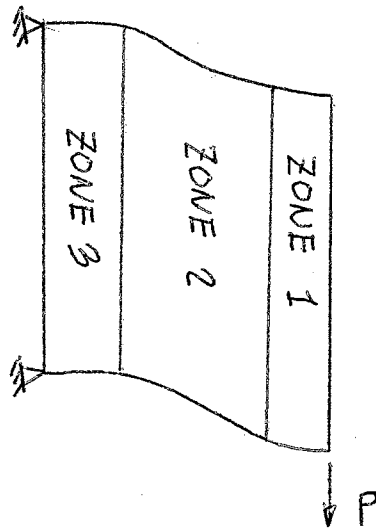


Figure 18. Four Corner
Fixed
Diaphragm

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- (6) Bryan, E. R. and W. M. Dakhakhni. "Shear of Thin Plates With Flexible Edge Members." Journal of the Structural Division, ASCE, Vol. 90, No. ST 4 (August, 1964), pp. 1-14.

APPENDIX

"STRUDL" COMPUTER INPUT LIST

Truss Configurations

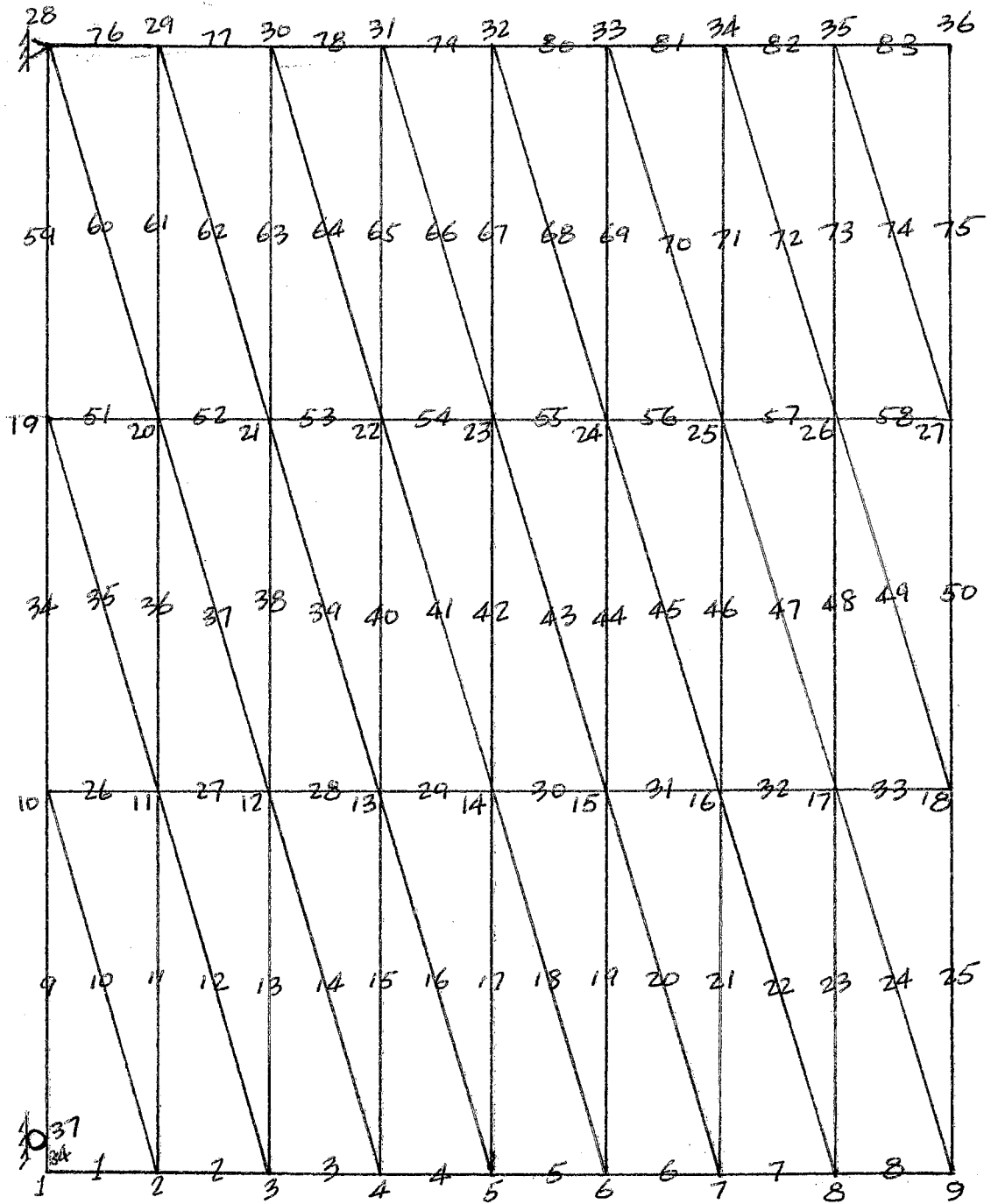


Figure 19. Typical Configuration of Truss Analogy

Joint Coordinates

<u>Joint No.</u>	<u>X</u>	<u>Y</u>	<u>Joint No.</u>	<u>X</u>	<u>Y</u>
1	0.0	0.0	33	120.0	240.0
2	24.0	0.0	34	144.0	240.0
3	48.0	0.0	35	168.0	240.0
4	72.0	0.0	36	192.0	240.0
5	96.0	0.0	37	0.0	4.0
6	120.0	0.0			
7	144.0	0.0			
8	168.0	0.0			
9	192.0	0.0			
10	0.0	80.0			
11	24.0	80.0			
12	48.0	80.0			
13	72.0	80.0			
14	96.0	80.0			
15	120.0	80.0			
16	144.0	80.0			
17	168.0	80.0			
18	192.0	80.0			
19	0.0	160.0			
20	24.0	160.0			
21	48.0	160.0			
22	72.0	160.0			
23	96.0	96.0			
24	120.0	120.0			
25	144.0	144.0			
26	168.0	168.0			
27	192.0	192.0			
28	0.0	0.0			
29	24.0	24.0			
30	48.0	48.0			
31	72.0	72.0			
32	96.0	96.0			

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171	MEM PROP PRISM AX 0.168 IZ 0.1
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217 MEM REL
218 10 STA MOM Z END MOM Z
219 11 STA MOM Z END MOM Z
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253 60 STA MOM Z END MOM Z
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266 73 STA MOM Z END MOM Z
267 74 STA MOM Z END MOM Z
268 76 STA MOM Z
269 83 END MOM Z
270 84 STA MOM Z

CARD

271 LOADING '1' '4.55'

272 JOINT LOAD

273 9 FOR Y -6.0

274 INACT MEM 42 19 21 40 44 63 65

275 STIFFN ANALYSIS

276 LIST FOR DISP ALL

277 FINISH

VITA

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