

A SYNTHESIS METHOD FOR ADAPTIVE CONTROL  
OF NONLINEAR SYSTEMS WITH APPLICATION  
TO A ONE DEGREE-OF-FREEDOM  
MOTION SIMULATOR

By

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## LIST OF SYMBOLS

A	An nxn System Matrix
$a_i$ (i=1,4)	Orifice Area (in section 4.1)
$A_i$ (i=1,2)	Actuator Area (in section 4.1)
B	An nxm Distribution Matrix of Inputs
g	Gravitational Acceleration (381 in/sec <sup>2</sup> )
I	Control Input to Servo Valve (ma)
J	Performance Cost Function
K	Gain Matrix
$P_i$ (i=1,2)	Pressure (psi)
$P_v$	Pressure Drop Across Valve (psi)
$P_m$	Load Pressure (psi)
p	A Co-state Vector in Optimal Control Theory
Q	A Symmetric Positive Weighting Matrix
$Q_i$ (i=1,2)	Volumetric Flow (in <sup>3</sup> /sec)
R	A Symmetric Positive Weighting Matrix
r (or R)	System Input Vector
T,t	Time Variable
U or u	Total Control Vector
$U_i$ (i=1,m)	A Component of the Control Vector
u*	Optimal Control
$V_i$ (i=1,2)	Volume of Actuator Chamber
X	Total Space Vector

$x_i$ ( $i=1,n$ )	A Component of the State Vector
$\hat{X}$	Estimated Model State Vector
$\dot{X}_L$	A Velocity Limit of the Actuator
$X_L$	A Position Corresponding to $\dot{X}_L$
$X_{max}$	Maximum Allowed Displacement of Actuator
$Z$	Measurement Vector
$\alpha$	A Scalar Parameter
$\beta$	Bulk Modulus of Fluid ( $\text{Lb/in}^2$ )
$\epsilon$	A Scalar Parameter or Error
$\rho$	Density of Fluid
$\phi$	Analytic Vector Function (Section 1.3.2.1) or System Parameter Vector Elsewhere
$\hat{\phi}$	Estimated Parameter Vector
$\omega_n$	Undamped Natural Frequency of a System (rad/sec)

## CHAPTER I

### INTRODUCTION AND BACKGROUND

Classical control system design is generally a trial-and-error process in which various methods of analysis are used iteratively to determine the design parameters of an "acceptable" system. A new and direct approach to the synthesis of complex, integrated control systems has been made feasible by advances in digital computers.

The application of modern control theory to the synthesis of systems that include electro-hydraulic servo components has been limited. Hydraulic subsystems have typically been designed separately from the rest of the system, without regard to impact on overall system performance.

This thesis discusses a technique for the design of on-line adaptive controllers for nonlinear systems which incorporate electro-hydraulic drives. Special emphasis is given to the development of an integrated control system for a one degree of freedom motion platform in which the highly nonlinear equations of motion form a part of the plant model.

#### 1.1 Motion Platform

A "motion platform" provides motion and force infor-

mation to the trainee in an aircraft training simulator. The operating envelope of the ground-based motion platform is limited by economic constraints and technical considerations. Usually the system is capable of providing moderate "g" cues in the frequency band of 0.5 Hz to 4.0 Hz. It is also capable of providing useful acceleration information for maneuvers with periods of 2 seconds or more by tilting the simulator. This system is very limited in evoking the physiological effects due to high-g maneuvers. On the other hand, it is quite useful for providing vibration or short duration alerting cues. A detailed discussion on the tasks of the motion platform and how these tasks are related to the human perceptual system can be found in Reference 1. A typical, constrained, motion platform response to aircraft dynamics inputs is delineated in Figure 1. The control provides the higher frequency "onset" portion of the acceleration profiles, then returns the cockpit to a neutral position, so as to be able to display the next "onset" cue. This process is known as "washing out" the motion cues. Washout levels should be constrained to values below the trainee's indifference threshold (from Reference 1: 2 degrees/sec and 0.1g for the semicircular canals and otoliths, respectively).

The common motion platform is driven hydraulically and is controlled via a conventional electro-hydraulic servo system. Signal inputs from aircraft equations of motion

(accelerations or rates or positions according to the specific configuration of the simulator) are processed via filters and directed to the servo controller. A typical block diagram of a motion platform control system is shown in Figure 2.

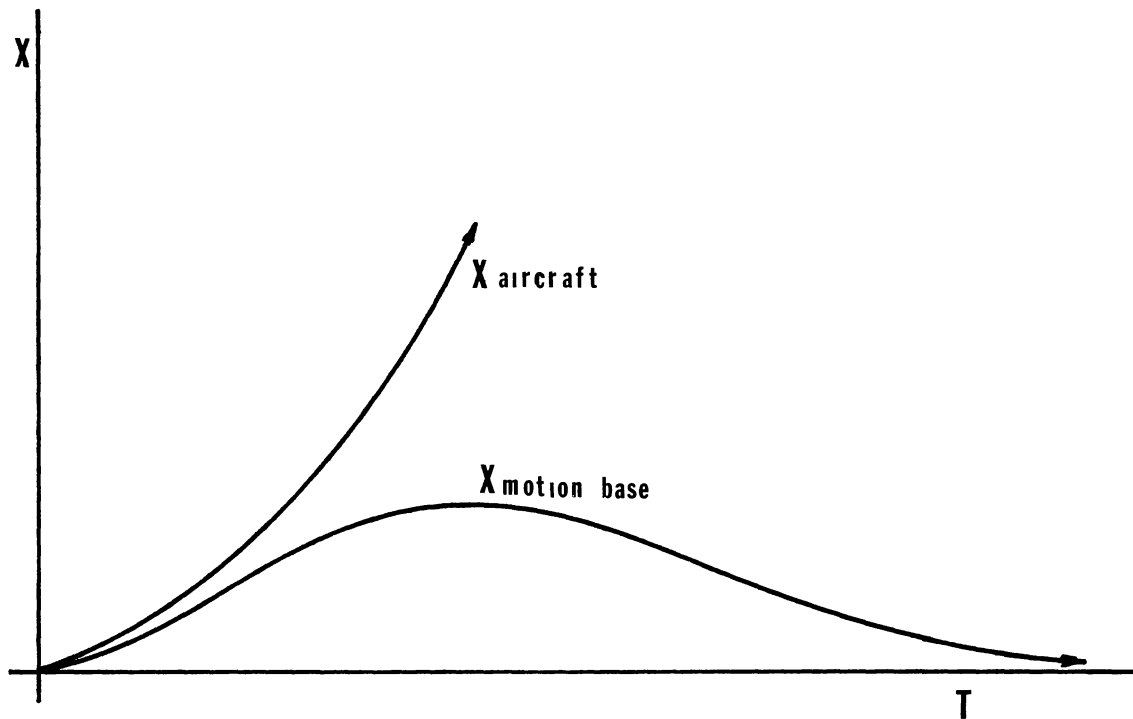


Figure 1. Typical System Response with Washout

New, modern approaches to the design of a motion platform control system (which omit the model of the hydraulic

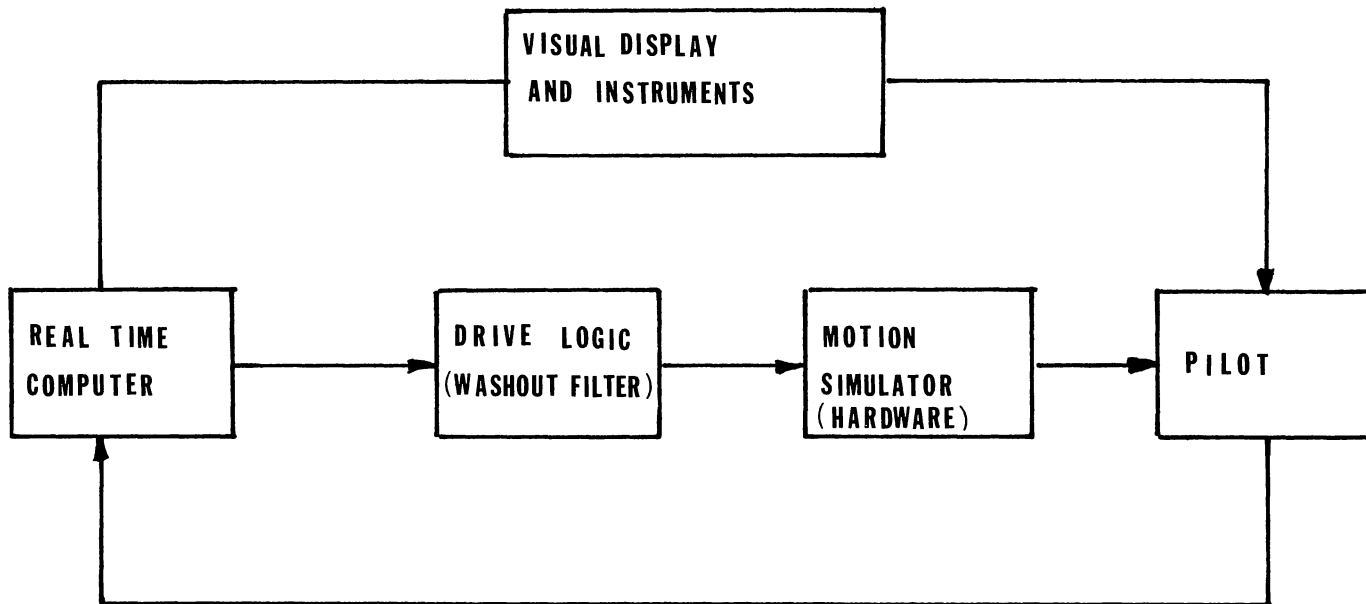


Figure 2. Block Diagram of Moving Base Simulator  
(From Ref.11)



system) are documented in References 2-11. A military standard of motion system requirement is given in Reference 12.

## 1.2 The Role of Digital Computers in Closed Loop Electrohydraulic Control Systems

R.H. Maskrey (13) explored possible applications of microcomputer technology for expanding the utilization of electrohydraulic servos. Possible uses of computer techniques can be categorized into six areas:

- a. Closing the loop. Using the computer as a closed-loop servo-controller.
- b. Pre-loop processing. Processing command information ahead of a conventional closed-loop servo.
- c. Peripheral processing. Where information is processed both ahead of and subsequent to a conventional closed-loop servo.
- d. Adaptive control. Using intelligence to modify the basic closed-loop control process.
- e. "Smart" Redundancy. Improving a redundant servo with the addition of higher level thinking.
- f. Improved time-optimal control. Enhancing bang-bang control.

Areas (b), (c) and (d) above are related to this

thesis. These areas are discussed in more detail below.

### 1.2.1 Pre-loop Processing

Computers are in use today as pre-loop processors or as command generators for conventional analog servos. This approach is commonly used in motion platform control. The computer is used effectively to perform interface functions, to interpret keyboard readings, to process rate signal inputs, or to monitor interactions between subsystems. The computer outputs drive a high performance servo.

### 1.2.2 Peripheral Processing

"Peripheral Processing" involves the use of a computer to handle related information both ahead of and after a conventional closed loop. This terminology should not be confused with the computer world's use of satellite computers to do 'peripheral' data processing. An example of peripheral processing which uses a large computer with a six degrees of freedom motion simulator is given in Reference 2. As seen in Figure 3, the computer processes the information of the flight simulation, manipulates the centroid transformation, washout system and actuator extension transformation. The computer also works with output information generated as a result of the servo operation, performs inverse transformation and predicts the position limits. Based upon these calculations, the computer is able to use

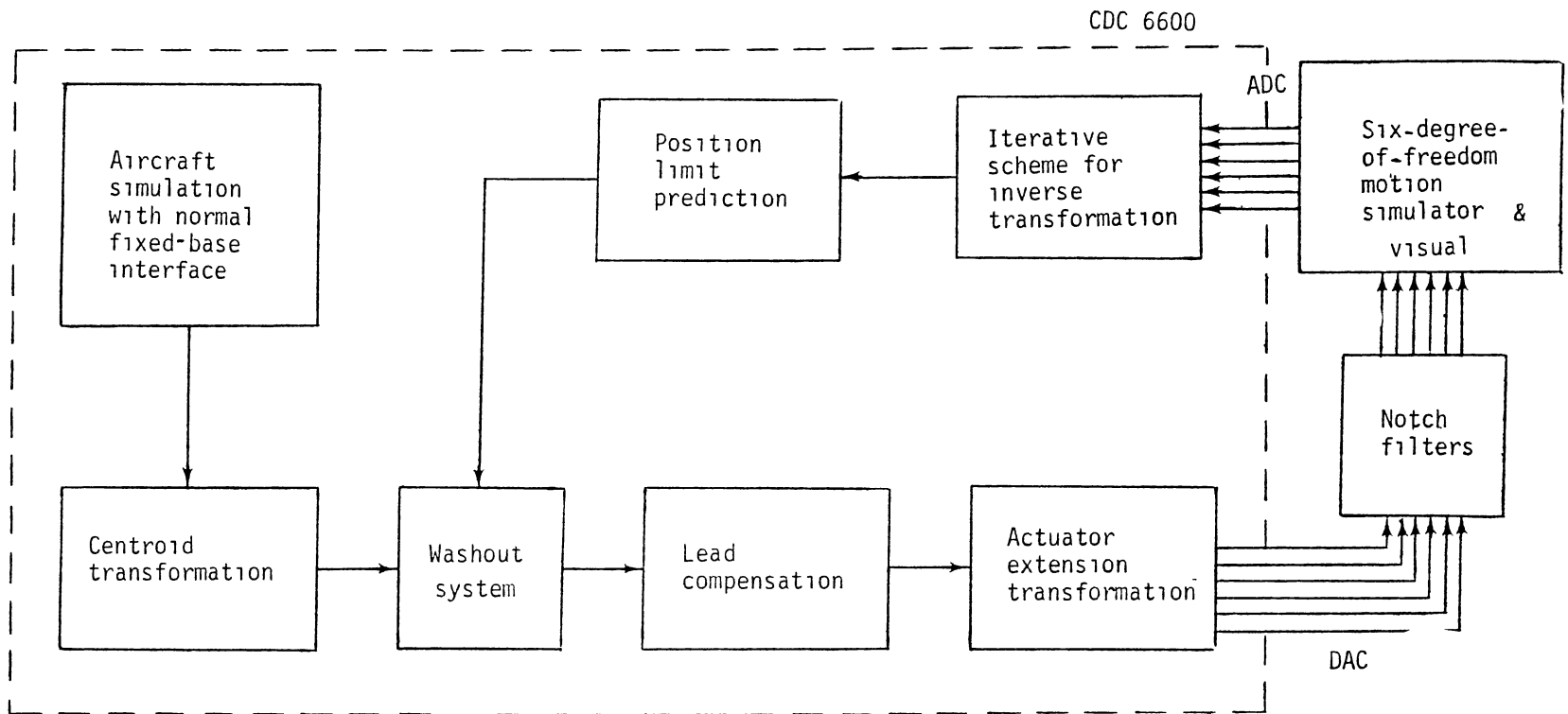


Figure 3. Major Blocks of a Real-Time Motion Simulator (From Ref.2)

the output information to improve the next cycle. Other cases of peripheral processing have been discussed by R. L. Kosut (6) and W. R. Sturgeon (10). In these cases, optimal control theory is used to generate an optimal controller (i.e., linearized model with constant feedback gains) based on information about the platform dynamics. But, in each of these cases, the traditional hydraulic servo system remains in the loop, untouched.

### 1.2.3 Adaptive Control

If there is a "best" application of digital computers to electrohydraulic control, then adaptive control may be it. It has long been held that adaptive control requires the capabilities of a computer. A self adaptive control system is one that is able to adjust itself to provide optimal or, perhaps, consistent control in the presence of identifiable changes. Typically, adaptive control should be used where constant performance is desired even though the following changes occur.

- a. Changes in the physical process being controlled with respect to time.
- b. Changes that occur in the power availability, such as changes in hydraulic supply pressure due to other demands.

- c. Changes in loading conditions, such as load inertia changes because of normal operation.
- d. Changes of gain with amplitude.

As a result of these changes, it may be desirable to control a variety of conditions based upon historical and predicted performance of the system. These conditions could include limits, gains, boundaries for nonlinearities and even control modes.

### 1.3 Control System Synthesis

The design of an automatic control system generally involves the selection of additional components which usually have adjustable parameters such that the overall system meets a desired performance specification. For example, this performance specification may be formulated (performance index) in terms of the minimization of an error criterion, a settling time, an energy constraint, or, it may be simply required that the response be stable.

The performance index as frequently used in control system design provides a quantitative measure of system performance and is normally chosen to emphasize important system characteristics. This quantitative measure is very important for parameter identification, state estimation, and for the design of optimal and suboptimal control systems.

### 1.3.1 Feedback Control Synthesis for Deterministic Linear Systems

The equations governing the behaviour of dynamic systems of the type considered in this thesis are usually non-linear. Even in cases where a linear approximation is justified, its range of validity is likely to be limited. In this thesis nonlinear equations of motion will be used exclusively. However, in order to start the iterative computation in defining the control laws, there is a need to start with a stable control law before the first iteration is executed. This control law is calculated for the linearized system operating around a steady-state equilibrium position.

Generally, the control law for the linearized system can be calculated (provided the system is controllable) by using one of the numerous methods of classical or optimal control.

In this thesis, the excess pole specification method of C.W. Merriam (14) is used to generate the linear system control law.

### 1.3.2 Control Synthesis for Nonlinear Systems

General approaches for controller synthesis similar to those available for linear systems do not exist at the present time. A few of the proposed approaches to solve the above problem are reviewed in this section.

1.3.2.1 A Functional Expansion Approach to the Solution of Nonlinear Feedback Problems. Methods for synthesizing suboptimal feedback control laws for nonlinear systems based on a functional expansion technique have been proposed (for example, see References 18 and 19). These methods apply to controllable dynamical systems modelled by

$$\dot{X} = AX + \varepsilon\bar{\Phi}(X) + BU \quad X(0) = X_0 \quad (1.3.2.1-1)$$

where  $X$ , an  $n$ -vector, is the state;

$U$ , an  $m$ -vector, is the control;

$X_0$  is the initial state;

$\bar{\Phi}$  is an analytic vector function;

$A$  and  $B$  are constant matrices, and  $\varepsilon$  is a scalar parameter.

W.L.Garrard (18) suggested that the optimization problem is the determination of a feedback control which minimizes the index of performance

$$J = \frac{1}{2} \int_0^{\infty} (X'QX + U'RU)dt \quad (1.3.2.1-2)$$

$Q$  and  $R$  are symmetric positive definite matrices. The optimal control  $U^*$  minimizes the scalar function

$$H(X,p,U,\varepsilon) = p'(AX + \varepsilon\bar{\Phi}(X) + BU) + \frac{1}{2}(X'QX + U'RU) \quad (1.3.2.1-3)$$

and is given by

$$U^* = -R^{-1}B'p \quad (1.3.2.1-4)$$

where the costate equations are:

$$\dot{p} = -\frac{\partial H}{\partial X} = -A'p - QX - \epsilon\Phi'_X p \quad (1.3.2.1-5)$$

$$\lim_{t \rightarrow \infty} p(t) = 0$$

Garrard proposed the block diagram of Figure 4 for control. Garrard noted that this scheme allows the designer to easily calculate a second order approximation to the optimal control. He also stressed that this control is only useful in some domain of asymptotic stability surrounding the origin.

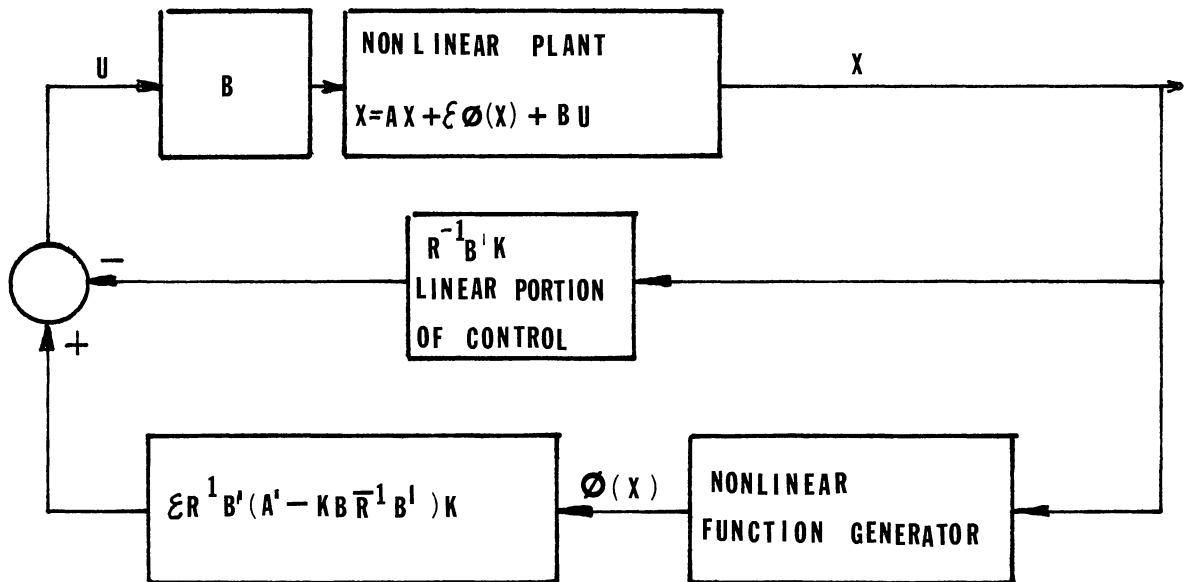


Figure 4. An Implementation of the Suboptimal Control  
(From Ref 18)



1.3.2.2 Optimal Control Law. The optimal controller synthesis problem can be solved utilizing Pontryagin's Maximum Principle (see also Reference 15).

Consider a controlled dynamic process given by a set of first-order differential equations:

$$\begin{aligned} \dot{x}_i &= f_i(X, \phi, U, T) & (1.3.2.2-1) \\ i &= 1, 2 \dots n \end{aligned}$$

where,  $x$  = states,  $\phi$  = system parameters,  $U$  = control,  $t$  = time; and a performance index

$$J = \int_0^T L(X, U, r, t) dt \quad (1.3.2.2-2)$$

which is to be minimized. The integral  $L(X, U, r, t)$  is the cost function, and represents a measure of instantaneous change from the ideal performance. The maximum principle requires that the optimal control,  $U^*$  which minimizes  $J$  will maximize the scalar Hamiltonian function:

$$H(X, U, p, t) = \sum_{i=1}^n p_i f_i(X, U, \phi, t) + L(X, U, r, t) \quad (1.3.2.2-3)$$

where vector  $p$ , called the costate vector, is given by:

$$\begin{aligned} \dot{p}_i &= - \frac{\partial H(X, U, p, t)}{\partial x_i} & (1.3.2.2-4) \\ i &= 1, 2 \dots n \end{aligned}$$

and

$$\dot{x}_i = \frac{\partial H(X, U, p, t)}{\partial p_i} \quad (1.3.2.2-5)$$

The optimal trajectory  $x^*(t)$  and  $p^*(t)$  are found by solving the above equations with  $2n$  boundary conditions. The optimal control  $U^*(t)$  is obtained in terms of  $x$ ,  $p$  and  $t$  as:

$$U^* = U(x^*, p^*, t)$$

The individual optimal control solution is dependent upon the particular choice of performance index.

The need for the solution of two point boundary value problems (TPBVP) has attracted a lot of attention in recent years. Nonlinear TPBVP's are encountered in solving optimal control problems via the maximum principle. Numerical solution of the optimal control problems may be obtained using iterative or non-iterative techniques. These methods suffer from the disadvantage however, that they yield the optimal control in the open loop form. In some cases, the problem under consideration may be sensitive to small perturbations in the initial costate and, as a result, convergence to an optimal solution may be slow if convergence occurs at all.

1.3.2.3 Direct Control Algorithm. Although a controller designed by the optimal control approach is optimal in the sense of satisfying the necessary conditions for optimality, the resultant control system is open loop and,

as such, may be very sensitive to environmental disturbances. Thus, it seems appropriate to consider methods for designing a closed-loop control system to overcome such problems. In their paper (20) H. Kaufman and R. Teavassos suggested the following method. Suppose the controller is constrained to be of the form:

$$U(t) = h[X(t),t] = KX(t) \quad (1.3.2.3-1)$$

where  $K$  is a gain matrix whose elements must be determined. The problem is to find a finite number of constants  $K_i$ , which minimize:

$$J = \Phi[x(t_f),t_f] + \int_{t_0}^{t_f} L[x(t),h(x(t),t)]dt \quad (1.3.2.3-2)$$

subject to the dynamic constraint given by

$$\dot{x}(t) = f[x(t),h(x(t),t),t] \quad (1.3.2.3-3)$$

Kaufman and Teavassos developed and evaluated control computation algorithms that employ a significant degree of parallelism. Their work involved the evaluation of two approaches: optimal control and direct suboptimal control. They recommended the direct estimation algorithm should be used only if no appreciable noise exist. (No solution was given for this problem). On the other hand, if process noise is excessive, then the indirect method (optimal, two

point boundary value problem in which the process noise is regarded as control) should be considered.

With regard to the control algorithms, Kaufman and Teavassos recommended the use of the direct gain optimization procedure in view of their findings that near optimal response was obtained without an excessive amount of computation. Also, they concluded that this procedure should be seriously considered because the equations of motion of a highly nonlinear system can be utilized directly in the control system design process.

## 1.4 Algorithm for Unconstrained Minimization

From the material discussed in previous sections it is evident that an efficient algorithm for minimization of a nonlinear function, called a performance index, must be employed. The optimization problem will be of the form:

$$\text{minimize } f(x), \text{ subject to } x \in \Omega$$

where  $f$  is a real-valued function and  $\Omega$ , the feasible set, is a subset of  $E^n$ . In the completely unconstrained case,  $\Omega = E^n$ .

Usually, optimization problems have constraints, like the differential constraints ( $\dot{x} = f(x, t)$ ) of the dynamic system; however, some of the most powerful and convenient methods of solving constrained problems involve the conversion of the problem to one of unconstrained minimization. A comprehensive discussion of nonlinear optimization can be found in References 22 and 23. A short overview of this subject is given in Appendix A, including a description of Davidon's optimization algorithm (without line search) that was used in this research.

### 1.4.1 Parallel Algorithms for Reducing Computation Time

The computer industry has developed parallel computers which are capable of performing the same set of instructions on many data sets simultaneously. Basically, a parallel

computer can be viewed as a set of processing elements, each of which has its own local memory and a repertoire of arithmetic and logical instructions. The role of the central processor is to coordinate the efforts of each processing element while its local memory is used for temporarily storing intermediate results.

Due to the availability of parallel processors, a number of nonlinear estimation and control algorithms (for parallel processing) have been proposed. Applications of quasi-Newton algorithms (for parallel processing) are reviewed in Reference 20, while work on square-root algorithms for optimal estimation (linear systems) is reported in Reference 26.

### 1.5 Motivation for the Research

Electrohydraulic subsystems normally are designed separately from the rest of the system, and then experimentally tuned to give the "best" performance for a specific limited operational envelope. References 1-11 ignored the role of the electrohydraulic servo dynamics in the control system design. R.L.Kosut (6) recommended the inclusion of actuator dynamics in the system model when the controller is designed analytically.

There is a need for an integrated control system design scheme that includes the hydraulic subsystem dynamics in the overall plant model. This subsystem can be modelled in its

natural state, that is using variables such as pressures, velocities and displacements, that are readily available for measurement.

The electrohydraulic servo system is highly nonlinear, with parameters that can only be assumed e.g bulk modulus of fluid, valve gain which changes near saturation, is also different for pressure port and return port if the actuator is unsymmetrical. A procedure is needed to determine a control law which is adaptive in nature, so that performance is consistent regardless of the nonlinearities and drifts in parameter values.

Another important design consideration is the washout motion for the specific application. If the computer is used for on-line control, it may perform decisions based upon state measurement and known hardware limitations and generate the inputs required for washout motion.

## 1.6 Research Objectives and Contributions

The objectives of this research were as follows:

1. To develop computationally efficient procedures for on-line identification and control of nonlinear dynamical systems with time varying inputs.
2. To formulate a washout strategy for a one degree of freedom motion platform.

3. To develop a state-space model of the motion platform system that will include the nonlinear actuator dynamics.
4. To simulate the proposed control system off-line for various inputs and modes of operation (e.g. follower mode, washout mode) and to evaluate the results.

The principal results from conducting this research are:

1. A strategy has been developed for the adaptive control of nonlinear systems using preselected integration intervals. (Objective 1)
2. The Q-N optimization method without line search has been shown to be an efficient method for solving the dynamical optimization problem (nonlinear identification and control). (Objective 2)
3. A braking/washout procedure can produce software generated inputs to the system in such a way that cues will not be attenuated unless physical constraints of the actuator dictates this attenuation. (Objective 3)
4. It has been demonstrated that the inclusion of the nonlinear hydraulic subsystem dynamics in the plant model produces a control system design with improved overall system performance, compared with a design which assumes the hydraulic subsystem dynamics are negligible. (Objectives 3 and 4).



## CHAPTER II

### METHOD AND PROCEDURES

In this chapter, an adaptive control algorithm is given, which can be used to solve nonlinear identification and control problems using digital computers. Section 2.1 presents the proposed control method, Section 2.2 presents the identification procedure and Section 2.3 describes the approach chosen for the calculation of inputs to the washout motion.

The direct identification and the direct control methods were chosen in lieu of optimal control and estimation approaches for the reasons discussed in Section 1.3.2.3. Also, Kaufman and Teavassos (20), showed that for a sample system, one iteration of the indirect (optimal) control algorithm required significantly more time to execute compared with the direct gain optimization procedure. This observation was also valid for the indirect and direct estimation algorithms.

#### 2.1 Nonlinear System Control Algorithm

This section discusses the development of a numerical method for synthesizing the controller for a nonlinear dynamical system. The approach is a suboptimal, direct

search for the closed-loop gains that will minimize an error function criteria.

### 2.1.1 Problem Statement

The physical process to be controlled is assumed to be an n-th order continuous-time system:

$$\begin{aligned} \dot{x}_1 &= f_1[x_1(t), x_2(t), \dots, x_n(t), t] \\ \dot{x}_2 &= f_2[x_1(t), x_2(t), \dots, x_n(t), t] \\ & \vdots \\ \dot{x}_n &= f_n[x_1(t), x_2(t), \dots, x_n(t), t] \end{aligned} \tag{2.1.1-1}$$

which is often expressed in vector form

$$\dot{x}(t) = f[x(t), t] \tag{2.1.1-2}$$

where  $x(t) \in \mathbb{R}^n$  is the state vector of the system.

The controller to be constrained is:

$$U(t) = K(t)[r(t) - x(t)] \tag{2.1.1-3}$$

where,

$U(t) \in \mathbb{R}^m$  is the control vector,  $m < n$

$K(t)$  is an  $m \times n$  gain matrix

$r(t) \in \mathbb{R}^n$  is the input vector signal

The problem is to calculate the values of the gain matrix elements  $k_{ij}$  that will minimize the quadratic error function:

$$J = \int_{t_0}^{t_0 + \Delta t} [x(t) - r(t)]^T [Q] [x(t) - r(t)] dt \quad (2.1.1-4a)$$

Or, for finite steps  $L$  of integration,

$$J = \sum_{j=1}^L [x(j) - r(j)]^T [Q] [x(j) - r(j)] \quad (2.1.1-4b)$$

Subject to the dynamic constraints,

$$\begin{aligned} \dot{x} &= f[x(t), U(t), \hat{\phi}, t] & (2.1.1-5) \\ x(t_0) &= z(t_0) \end{aligned}$$

$z(t)$  is the plant state vector (measurement data), and  $\hat{\phi}$  is the parameter vector of the nonlinear system.

### 2.1.2 Control Algorithm

The solution of the above problem requires the definition of the input to the system,  $r(t)$ , and the weighting matrix  $[Q]$  for the quadratic error function defined in the equation (2.1.1-4).

Since  $x(t_0)$  and  $r(t_0)$  are known, but  $r(t)$  is, in general time dependent, there is a need to extrapolate  $r(t)$  using past information. The approach used is as follows:

Extract from storage the last 4 reference input values of the previous cycle (see also Figure 5). Calculate the first, second and third order time derivatives by Newton's backward difference formula:

$$\begin{aligned}
 r'(i) &= (3r_i - 4r_{i-1} + r_{i-2})/2\Delta h \\
 r''(i) &= (r_i - 2r_{i-1} + r_{i-2})/\Delta h^2 \\
 r'''(i) &= (r_i - 3r_{i-1} + 3r_{i-2} - r_{i-3})/\Delta h^3
 \end{aligned}
 \tag{2.1.2-1}$$

Using these values, the input to the system can be predicted by using a Taylor Series Expansion:

$$\begin{aligned}
 r(t) = r(t_i+h) &= r(t_i) + hr'(t_i) + \left(\frac{h^2}{2}\right)r''(t_i) \\
 &+ \left(\frac{h^3}{6}\right)r'''(t_i)
 \end{aligned}
 \tag{2.1.2-2a}$$

or

$$\begin{aligned}
 r(i+k) &= r(i) + k\Delta hr'(i) + \frac{(K\Delta h)^2}{2} r''(i) \\
 &+ \frac{(K\Delta h)^3}{6} r'''(i)
 \end{aligned}
 \tag{2.1.2-2b}$$

Using this procedure, the integration should be performed over a fraction of the update interval.

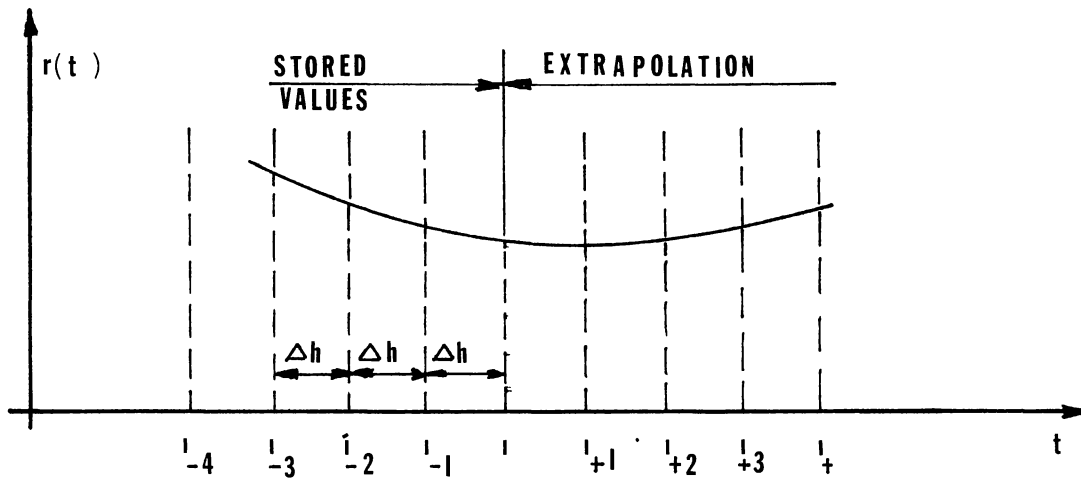


Figure 5. Reference input extrapolation for Control Law Synthesis.

Differentiating the quadratic error function defined in (2.11-4)

$$\frac{dJ}{dt} = [x(t) - r(t)]^T [Q] [x(t) - r(t)]$$

and rearranging for a diagonal matrix  $Q$

$$\frac{dJ}{dt} = \sum_{i=1}^n Q_i (x_i - r_i)^2 \quad (2.1.2-3)$$

If the following values are set

$$\begin{aligned}
 Q_i &= 1.0 & (i = k) \\
 Q_i &= 0.0 & (i = k), \qquad (2.1.2-4)
 \end{aligned}$$

then a specific mode of control is obtained. If the subscript  $k$  corresponds to the velocity ( $x(k) = \text{velocity}$ ), then the system will follow the velocity input only. If the subscript  $k$  corresponds to the position input, a position servo will be obtained. This mode selection for control is very flexible because it does not require a different algorithm for each mode of operation selected.

### 2.1.3 Implementation Steps

The control synthesis procedure can be summarized in the following steps.

#### Step (0). Off-line Initialization

Linearize the open-loop system equations of motion with nominal system parameters around a steady-state equilibrium point. Calculate  $K_{\text{last}} = K(0)$  for the linearized deterministic system using any of the known methods so that the forward integration of Equation 2.1.1-5 is stable.

#### Step (1). On-line

Apply the input to the system. Record the input signal for the last four grid points. Calculate the first, second and third time derivatives for the input signal at the last point of the measured data.

Step (2). On-line

Given  $K_{\text{last}}$  and the initial state  $x(i)$  as well as  $r(i+j)$  from extrapolation, use a nonlinear minimization procedure to update  $K$  so that the cost function  $J$  in Equation 2.1.1-4b will be minimized.  $x(i+j)$ ,  $j=1,L$  can be calculated by the integration of Equation 2.1.1-5 forward in time over the time interval  $(t_i, t_{i+L})$  from the initial state  $x(i)$ .

Step (3). On-line

If  $|j^{n+1} - j^n| < \epsilon$ , where  $n$  denotes the number of optimization iterations performed, or if the allowed number of optimization iterations was reached, accept the new value of  $K$  and update the gain coefficient as follows:

$$K_{\text{new}} = K_{\text{last}} + \alpha(K - K_{\text{last}}) \quad (2.1.3-1)$$

$$0.0 < \alpha < 1.0$$

Then return to step (1).

## 2.2 Nonlinear System Parameter

### Identification

This section discusses the development of a numerical method for parameter identification of nonlinear dynamical system. The approach taken was a direct search for the parameters by minimization of a specific cost function. This specific cost function is a measure of the deviation of the

response between the plant states (measurements) and the assumed model (unknown parameters) states.

### 2.2.1 Problem Statement

The nonlinear dynamical system model and the measured variables may be represented by the state-vector equations:

$$\dot{x} = f[x(t), U(t), \phi(t), t] \quad (2.2.1-1)$$

$$z(t) = h[x(t), t] + w(t),$$

where  $x(t) \in R^n$  is the state vector.

$z(t) \in R^L$  is the measurement vector.

$U(t) \in R^m$  is the control vector.

$\phi(t) \in R^k$  is the parameter vector.

$w(t) \in R^L$  is a zero-mean white noise sequence.

Assuming that all the states are accessible for measurement, the problem reduces to searching for the values of the parameters that will minimize the error function:

$$J = \int_{t_0 - \Delta t}^{t_0} [z(t) - \hat{x}(t)]^T [Q] [z(t) - \hat{x}(t)] dt \quad (2.2.1-2)$$

subject to the dynamic constraints of the assumed plant model:

$$\dot{\hat{x}}(t) = f(\hat{x}(t), U(t), \hat{\phi}(t), t) \quad (2.2.1-3)$$



where  $\hat{x}(t_0 - \Delta t) = Z(t_0 - \Delta t)$ , or, for finite steps  $L$  of integration,

$$J = \sum_{j=1}^L [Z(j) - \hat{x}(j)]^T [Q] [Z(j) - \hat{x}(j)] dt \quad (2.2.1-4)$$

Working directly with the nonlinear system equations of motion rather than trying to linearize the system may render faster system identification. Usually only a few parameters of the nonlinear system need identification. If the system is represented locally as  $\dot{x} = [A]X + [B]U$ , then the number of parameters to be identified will grow. If there are  $n$  states and  $m$  inputs, then the  $[A]$  matrix with  $n \times n$  elements and the  $[B]$  matrix with  $n \times m$  elements need identification. For example, if  $n = 4$ ,  $m = 1$  then  $4 \times 4 + 4 \times 1 = 20$  parameters need to be identified. Figure 6 depicts how the sampling and integration intervals are related. The identification process is applied only during a fraction of the interval between updates, thereby conserving function evaluation time.

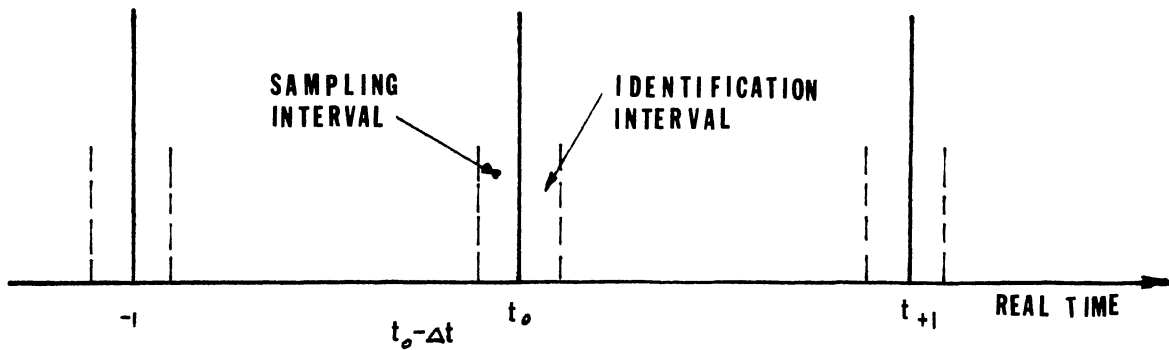


Figure 6. Integration and Sampling Interval for the Identification Process.

### 2.2.2 Implementation Steps

Step (0). Off-line Initialization

Develop the nonlinear model of the system:

$$\dot{\hat{x}}(t) = f[\hat{x}(t), \hat{U}(t), \hat{\phi}(t), t] \quad (2.2.2-1)$$

where  $\hat{U}(t) = K[r(t) - \hat{x}(t)]$

Calculate and initialize the nominal vector of system parameters  $\hat{\phi}$ .

Step (1). On-line

Apply the input to the system and record the input and measurements of the response (state values) during the measurement data window.

Step (2). On-line

Given  $\hat{\phi}_{\text{last}}$  and the initial state  $\hat{x}_i = z_i$ , use the nonlinear minimization procedure to update  $\hat{\phi}$  so that the cost function  $J$  in Equation 2.2.1-4 is minimized. The values of  $\hat{x}(i)$  in the cost function are obtained by forward integration of Equation 2.2.1-3.

Step (3). On-line

If  $|J^{n+1} - J^n| < \epsilon$ , where  $n$  denotes the number of optimization iterations performed, or if the allowed number of optimization iterations is exceeded, accept the new values of  $\hat{\phi}$  and update the system parameters as follows:

$$\hat{\phi}_{\text{new}} = \hat{\phi}_{\text{last}} + \alpha(\hat{\phi} - \hat{\phi}_{\text{last}}) \quad (2.2.2-2)$$

$$0.0 < \alpha < 1.0$$

Then return to step (1).

## 2.3 Washout Algorithm

The braking/washout procedure described in this section is related to a special mode of operation for the motion platform and involves the generation of a particular input to the control system.

### 2.3.1 Problem Statement

All motion simulators have limits on the amount of movement in each degree of freedom. The design of a system

which will transmit motion cues to a pilot while keeping the movement of the simulator within its constraints is the major task faced by the designer. After the initial cue has been transmitted, another function of the system is to return the simulator to its neutral position without the pilot being aware of the movement. This effect is termed "washout". This technique of keeping the simulator near its neutral position, maximizes the movement allowable for subsequent cues.

### 2.3.2 Algorithm

The signal input to the system should be switched from the follower mode to the washout mode in the translational degree-of-freedom at a safe point, so that the physical limit of the actuator is not reached.

$$\frac{1}{2} M \dot{X}_L^2 = \int_{X_L}^{X_{\max}} F dx = \int_{X_L}^{X_{\max}} M \ddot{X} dx, \quad (2.3.2-1)$$

where:

$\dot{X}_L$  is the velocity limit of the actuator at some position limit  $X_L$ ,

$X_L$  is the position corresponding to the velocity limit  $\dot{X}_L$ , and

$X_{\max}$  is the allowed maximum displacement of the actuator from neutral position.

If the maximum acceleration for washout is a constant  $\alpha$ , then

$$\frac{1}{2} \dot{X}_L^2 = \alpha(X_{\max} - X_L) \quad , \quad (2.3.2-2)$$

or

$$\dot{X}_L = \sqrt{2\alpha(X_{\max} - X_L)} \quad (2.3.2-3)$$

will define the switching point. Once the control mode is switched to washout, a command signal is generated to bring the motion base to the origin with acceleration level  $\ddot{x} \leq \alpha$ . When the base reaches the origin, the system is able to accept more inputs from the aircraft dynamics as produced by the real-time computations.

### 2.3.3 Implementation Steps (On-line)

Note: A "washout flag" is a switch that transfers the control system to work in washout mode when "True" and to follower mode when "False".

Step(1).

Evaluate state measurements of the system. If  $ABS(x) < \epsilon$ , then clear the washout flag. If  $\dot{x} \geq \dot{x}_L$ , set the washout flag. Here  $x$  is the translational displacement,  $\dot{x}$  is the translational velocity and  $\dot{x}_L$  is the velocity limit ( $\dot{x}_L = \sqrt{2\alpha(x_{\max} - x)}$ ).

If the washout flag is set, go to step (2). Otherwise, branch to the normal follower mode.

Step (2).

In the washout mode there are two regions of operation:

a) If the maximum displacement allowed is not reached, the system is in the brake mode.

b) If the maximum displacement allowed is exceeded, the system is approaching the steady-state position.

Step (2a).

The reference (acceleration) input to the system is:

$$\dot{r} = \dot{r} + K(\dot{x} - \dot{x}_L), \quad |\dot{r}| \leq \alpha \quad (2.3.3-1)$$

Step (2b).

If  $\dot{x} > \sqrt{2\alpha x}$  - the system moves too fast towards the origin, then

$$\dot{r} = \dot{r} + K(\dot{x} - \sqrt{2\alpha x}) \quad (2.3.3-2)$$

Otherwise,

$$\dot{r} = \dot{r} + K(\alpha - \dot{r}) \quad (2.3.3-3)$$

### Step 3.

An updated input  $r$  in the washout mode has been established; use it as a reference input to the system.

The Direct Identification method and the Direct Gain-Optimization method are known to offer the possibility of nearly-optimal control systems.

The contributions made in this chapter are related to the proposed algorithms. Integration intervals of the control algorithm (Section 2.1.2) and of the identification algorithm (Section 2.2.1) are related to a fraction of the update intervals. The updated values of gains and parameters are a function of previous results obtained during the iteration process and the new results obtained in the current iteration. The input to the system which is used for gain optimization, is extrapolated forward in time, using memorized data, rather than providing only a fixed value.

## CHAPTER III

### APPLICATION MODEL STUDIES

#### 3.0 General Considerations

Before a detailed analysis, synthesis and simulation of the application system can be undertaken, it is important that the system configuration and desired mode of operation be understood. So that this research will produce qualitative conclusions concerning the advantages of the method, the physical configuration and its dynamical characteristics should be typical of those now in use. Similar consideration should also be given to the hydraulic servomechanism and the measured system variables used to control it.

This section is concerned with a description of the selected one degree-of-freedom motion platform model, its lumped-parameter representation and the desired performance under controlled conditions.

For this investigation, a translational degree-of-freedom motion platform is considered. A recommended operational envelope, (from Reference 1), is given in Figure 7. This envelope is typical for the "heave axis" performance of motion platforms currently in service.

The motion platform system should work in two modes of operation: a follower mode and a washout mode. In the



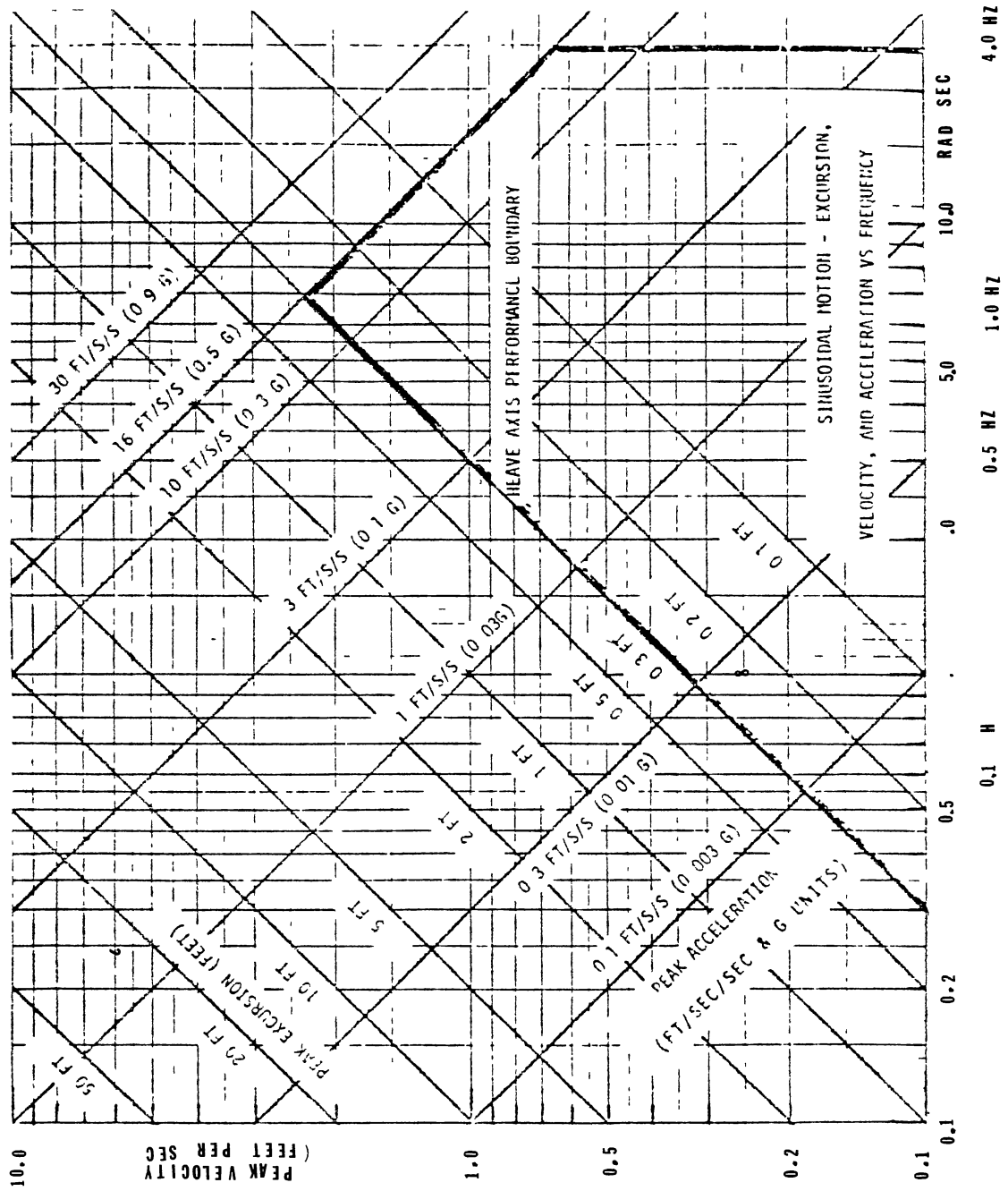


Figure 7. Vertical Axis Operational Envelope Recommended for Platform Motion Simulator. (From Ref. 1)

follower mode, the motion should follow the inputs from the aircraft dynamics as computed in real time by the computer. In case the platform measurements indicate that the physical limit of travel may be exceeded, the platform switches to the washout mode of operation. The latter mode is introduced to generate an input to the system independent of aircraft dynamics, in order to bring the platform to its neutral position.

A block diagram of the proposed system is depicted in Figure 8. Note that the aircraft dynamics model and the pilot model which appear on the block diagram are for descriptive and completeness purposes only. No attempt is made in this thesis to incorporate these models into the system under investigation. Furthermore, only one control variable is used, and it is assumed that complete state measurements are available.

### 3.1 Principles of Flow Control for Valve-operated Systems

Typically in hydraulic servo systems used in simulators, flow is controlled by throttling the fluid with the variable orifices of a control valve. If the flow rate demand is within its capacity, the pumping system normally acts as a source of substantially constant pressure.

The controlled flow rate is dependent on the pressure drops across the valve orifices, the density of the fluid,

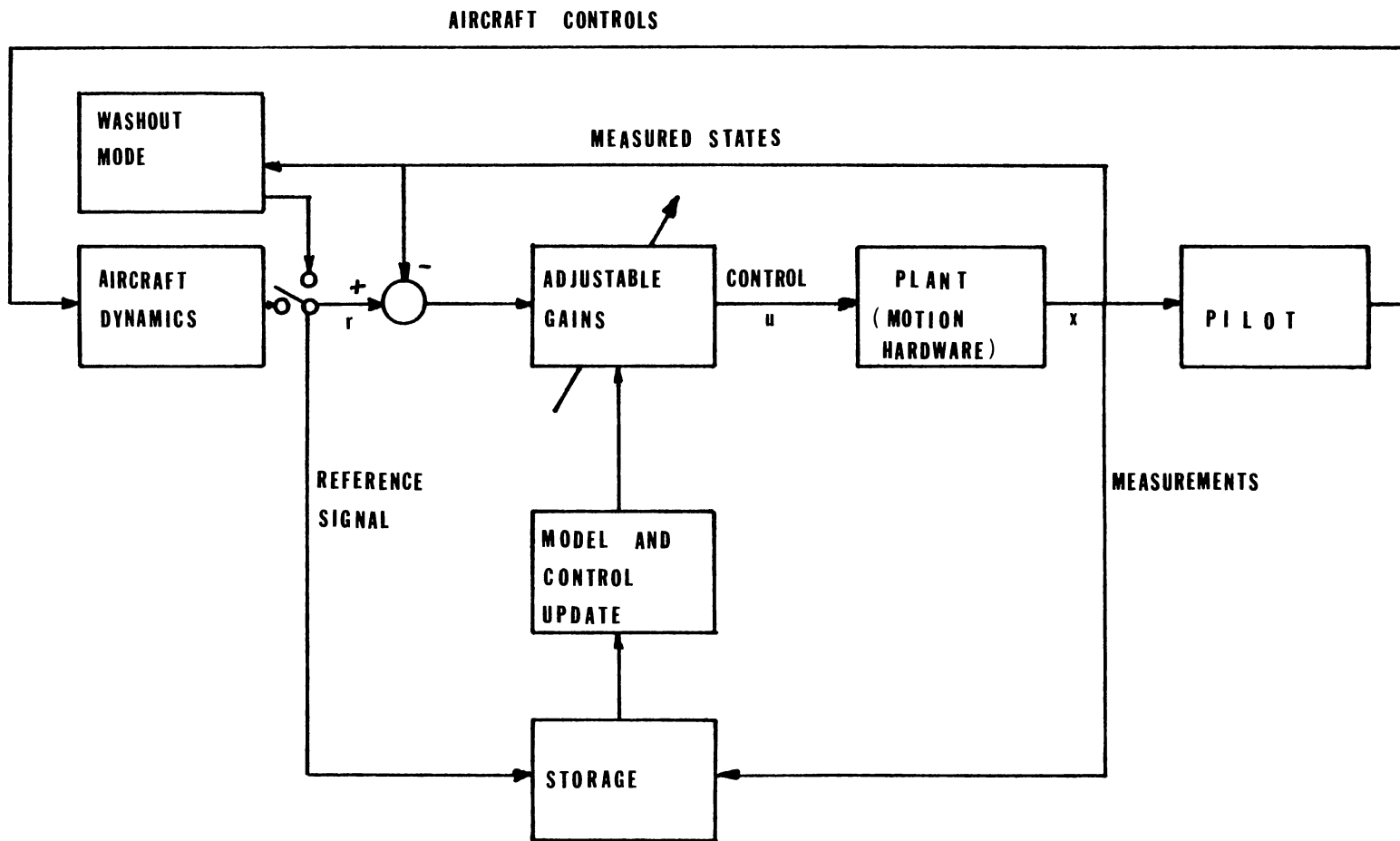


Figure 8. Proposed Control System for Motion Platform

the coefficients of discharge, and the orifice areas, and is relatively independent of the fluid temperature.

The general relationship between controlled flow rate, valve opening and pressure drop in a typical valve is:

$$q = C_d a \sqrt{\frac{2}{\rho}} \sqrt{\Delta P}, \quad (3.1-1)$$

where  $q$  is the flow through a given orifice,  $a$  is the orifice area,  $\Delta P$  is the pressure drop across the orifice,  $\rho$  is the mass density of the fluid and  $C_d$  is the coefficient of discharge.

The combination of four single orifices into the bridge arrangement shown on Figure 9, permits the modulation of fluid power to an actuator and load. The four variable orifices are coupled mechanically, so that:

$$a_1 = a_4 \quad \text{and} \quad a_2 = a_3$$

for a symmetrical valve,

$$a_{1\max} = a_{2\max} = a_{3\max} = a_{4\max}.$$

for a valve with negligible overlap or underlap (i.e. "line to line"),

$$\text{if } a_1 > 0, a_1 = 0 \quad \text{or} \quad \text{if } a_3 > 0, a_1 = 0.$$

Therefore, the flow path must be either through  $a_1$ , the Load and  $a_4$ , or through  $a_3$ , the Load and  $a_2$ .

The volumes of fluid in the actuator chambers are  $V_1$  and  $V_2$ . The volumetric flow rate into chamber 1 is  $Q_1$  the flow rate out of chamber 2 is  $Q_2$ , and the ram velocity is  $\dot{x}$ .

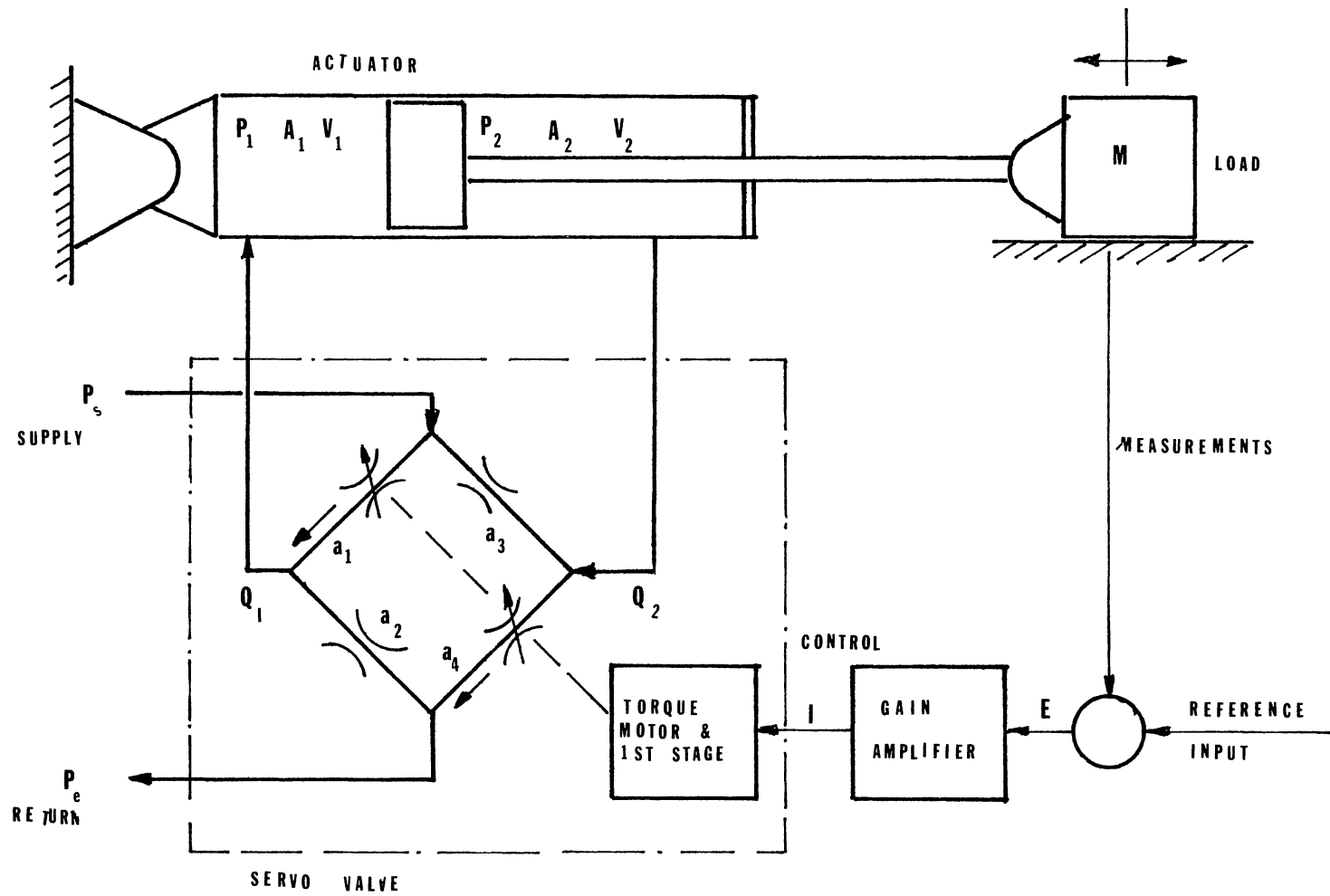


Figure 9. Typical Electrohydraulic Servosystem.

It is assumed that the pressure is uniform in each ram chamber and that no leakage occurs from chamber to chamber or past the piston rod.

The continuity equation applied to ram chamber 1 is:

$$\rho_1 Q_1 = \frac{d}{dt} (\text{mass}) = \frac{d}{dt} (\rho_1 V_1) \quad (3.1-2)$$

$$\rho_1 Q_1 = \dot{\rho}_1 V_1 + \rho_1 \dot{V}_1 .$$

Since the volume changes because of the ram motion, equation (3.1-2) may be written as:

$$Q_1 = \frac{\dot{\rho}_1 V_1}{\rho_1} + A_1 \dot{x} \quad (3.1-3)$$

From the definition of bulk modulus,

$$\frac{\dot{\rho}_1}{\rho_1} = \frac{\dot{P}_1}{\beta} \quad (3.1-4)$$

When equation (3.1-3) and equation (3.1-4) are combined

$$Q_1 = \dot{P}_1 \frac{V_1}{\beta} + A_1 \dot{x} \quad (3.1-5a)$$

Similarly, for ram chamber 2

$$Q_2 = -\dot{P}_2 \frac{V_2}{\beta} + A_2 \dot{x} \quad (3.1-5b)$$

### 3.2 Specific Hardware Configuration Chosen

A specific and typical hardware configuration was chosen to test the method. The actuator piston areas were chosen to be the same on both sides of the piston in order to simplify the model.

$$A = A_1 = A_2 = 8 \text{ in}^2.$$

Other parameters chosen were:

Actuator Displacement:	$X_{\max} = \pm 30.0$	in.
Load Mass:	$M = 8.635$	Lbf sec/in <sup>2</sup>
Supply Pressure:	$P_s = 2000.0$	lbf/in <sup>2</sup>
Fluid Bulk Modulus:	$= 150000.0$	lbf/in <sup>2</sup>
Servovalve:	Moog #78-14X; rated for 40 GPM at 1000 psi pressure drop, no load, 40 ma.	

#### 3.2.1 Nonlinear Equations of Motion (Open Loop)

From the chosen valve specifications:

$$Q = 0.1217 I \sqrt{P_v} \quad [\text{in}^3/\text{sec}] \quad (3.2.1-1)$$

$$|I| \leq 40\text{ma} \quad \text{for saturation.}$$

where  $P_v$  is the pressure drop across the valve,

$$P_v = P_s - P_m$$

and  $P_m$  is the load pressure

$$P_m = P_1 - P_2$$

$P_s$  is the supply pressure and  $I$  is the current input to the torque motor.

For a symmetrical valve and equal areas actuator:

$$P_s - P_1 = P_2 \quad ; \text{ assumes } P_e = 0$$

$$P_v = 2(P_s - P_1) = 2P_2$$

$$Q_1 = 0.172 I \sqrt{P_s - P_1}$$

$$Q_2 = 0.172 I P_2 \quad (3.2.1-2)$$

Taking into account the effects of negative current and back-pressure, equations (3.2.1-2) may be rewritten as:

$$Q_1 = 0.172 I \left[ \left| \frac{P_s}{2} + \left( \frac{P_s}{2} - P_1 \right) \frac{I}{|I|} \right|^{\frac{1}{2}} \cdot \text{sign} \left[ \frac{P_s}{2} + \left( \frac{P_s}{2} - P_1 \right) \frac{I}{|I|} \right] \right]$$

$$Q_2 = 0.172 I \left[ \left| \frac{P_s}{2} - \left( \frac{P_s}{2} - P_2 \right) \frac{I}{|I|} \right|^{\frac{1}{2}} \cdot \text{sign} \left[ \frac{P_s}{2} - \left( \frac{P_s}{2} - P_2 \right) \frac{I}{|I|} \right] \right] \quad (3.2.1-3)$$

From equations (3.1-5), the flow equations for the actuator:



$$Q_1 = \dot{P}_1 \frac{V_1}{\beta} + A\dot{X} \quad (3.2.1-4)$$

$$Q_2 = -\dot{P}_2 \frac{V_2}{\beta} + A\dot{X}$$

Neglecting the load friction (the friction force is less than 8 Lbf for moog Laminar actuators), the load equation is:

$$P_1 A - P_2 A = M\ddot{X} \quad (3.2.1-5)$$

The nonlinear equations of motion are obtained by combining equations (3.2.1-4) and (3.2.1-5).

Two system variables can be defined as follows:

$$X_1 = X, \quad X_2 = \dot{X}_1$$

Then the equations of motion are:

$$\begin{aligned} \dot{X}_1 &= X_2 \\ \dot{X}_2 &= \frac{1}{M} [P_1 A - P_2 A] \\ \dot{P}_1 &= \frac{\beta}{V_1} [Q_1 - AX_2] \\ \dot{P}_2 &= \frac{\beta}{V_2} [-Q_2 + AX_2] \end{aligned} \quad (3.2.1-6)$$

where  $Q_1$  and  $Q_2$  are defined in equation (3.2.1-3) and

$$V_1 = A(X_{\max} + X)$$

$$V_2 = A(X_{\max} - X)$$

### 3.2.2 Linearization of the equations of motion

The equations of motion of the model need to be linearized in order to find the required initial controller gains that will result in a stable system. The valve equations (3.2.1-2) are:

$$Q_1 = 0.172 I \sqrt{P_s - P_1}$$

$$Q_2 = 0.172 I \sqrt{P_2}$$

$$\text{Then } \Delta Q_1 = \left. \frac{\partial Q_1}{\partial I} \right|_i \Delta I + \left. \frac{\partial Q_1}{\partial P_1} \right|_i \Delta P_1$$

$$\text{and } \Delta Q_2 = \left. \frac{\partial Q_2}{\partial I} \right|_i \Delta I + \left. \frac{\partial Q_2}{\partial P_2} \right|_i \Delta P_2 \quad (3.2.2-1)$$

$$\text{or } \Delta Q_1 = 0.172 \left[ \sqrt{(P_s - P_1)}_i \Delta I - \frac{I_i}{2\sqrt{(P_s - P_1)}_i} \Delta P_1 \right]$$

$$\text{and } \Delta Q_2 = 0.172 \left[ \sqrt{(P_2)}_i \Delta I + \frac{I_i}{2\sqrt{(P_2)}_i} \Delta P_2 \right]$$

Taking the specific cases where  $I_i = 0.1 \times 40.0 = 4\text{ma}$  and  $P_{2_i} = (P_s - P_1)_i = 0.05 \times 2000.0 = 100 \text{ psi}$ , the linearized valve equations are:

$$\begin{aligned}\Delta Q_1 &= 0.172 (10\Delta I - 0.2\Delta P_1) \\ \Delta Q_2 &= 0.172 (10\Delta I + 0.2\Delta P_2)\end{aligned}\tag{3.2.2-2}$$

Using the given parameters and assuming  $V_1 = V_2 = 240 \text{ in}^3$ , the linearized equations of motion of the open-loop system are

$$\begin{aligned}\Delta \ddot{X} &= 0.926(\Delta P_1 - \Delta P_2) \\ \Delta \dot{P}_1 &= 625(\Delta Q_1 - 8\Delta \dot{X}) = 625(1.72\Delta I - 0.0345\Delta P_1 - 8\Delta \dot{X}) \\ \Delta \dot{P}_2 &= 625(-\Delta Q_2 + 8\Delta \dot{X}) = 625(-1.72\Delta I - 0.0345\Delta P_2 + 8\Delta \dot{X})\end{aligned}$$

Substituting

$$\begin{aligned}X_1 &= \Delta X \\ X_2 &= \Delta \dot{X} = \dot{X}_1 \\ X_3 &= \Delta P_1 \\ X_4 &= \Delta P_2 \text{ and } U = \Delta I,\end{aligned}$$

the resulting equations are

$$\begin{aligned}\dot{X}_1 &= X_2 \\ \dot{X}_2 &= 0.926X_3 - 0.926X_4 \\ \dot{X}_3 &= -5000X_2 - 21.5625X_3 + 1075U \\ \dot{X}_4 &= 5000X_2 - 21.5625X_4 - 1075U\end{aligned}\tag{3.2.2-3}$$

In the standard state vector representation, the linearized equations of motion are of the form

$$\dot{X} = AX + BU \quad (3.2.2-4)$$

In this example

$$A = \begin{bmatrix} 0.0 & 1.0 & 0.0 & 0.0 \\ 0.0 & 0.0 & 0.926 & -0.926 \\ 0.0 & -5000.0 & -21.5625 & 0.0 \\ 0.0 & 5000.0 & 0.0 & -21.5625 \end{bmatrix}$$

and

$$B = [0.0 \quad 0.0 \quad 1075.0 \quad -1075.0]$$

### 3.2.3 Calculation of Undamped Natural Frequency of Actuator Mass System

The selection of integration step sizes for on-line computation and system simulation is dependent to a large extent on the undamped natural frequency of the actuator-mass system. For the numerical values given above, the spring constant is

$$k_s = 2\beta A^2/V_0 = 2.150000.8^2/240 = 80000 \text{ Lb/in}$$

The undamped natural frequency of the system is

$$\omega_n = \sqrt{\frac{k_s}{m}} = 96.2 \text{ rad/sec}$$

or  $f_n = 15.3 \text{ Hz}$ , and the period is

$$T = \frac{1}{f_n} = \frac{1}{15.3} = 0.065 \text{ sec.}$$

This result was used to select the integration step size for off-line simulation (step size chosen was 0.001 sec.).

#### 3.2.4 Calculations of Position Control Gains for the Linearized Deterministic System

The excess pole specification method (14) and the computer programs that were provided by C. W. Merriam III, were used to calculate the control gains for the linearized deterministic system for this example. In order for the system to operate as a low pass filter up to 100 rad/sec (the undamped natural frequency of the open-loop system for the case considered is 96.2 rad/sec.), an appropriate closed-loop transfer function is

$$T_d(s) = \frac{1}{(1+0.01s)^3}$$

then

$$P(s) = T_d(s)^{-1} = (1+0.01s)^3 = 1.0 + 0.03s + 3.0 \times 10^{-4}s^2 + 1.0 \times 10^{-6}s^3,$$

so the P matrix is:

$$P_{(i,1)} = [1.0 \quad 0.03 \quad 3.0 \times 10^{-4} \quad 1.0 \times 10^{-6}].$$

Using the A and B matrices from equation (3.2.2-4) and selecting the H matrix (output:  $Y = HX$ ) as

$$H = [1.0 \quad 0.0 \quad 0.0 \quad 0.0],$$

which means position control, the following gain coefficients were obtained:

$$K = [502.228 \quad 10.4174 \quad .1295058 \quad -.1295058]$$

for the control law:

$$I = U = [K]\{r-X\} .$$

### 3.3 Simulation Results: Deterministic System

In this section, the simulated responses of linear and nonlinear deterministic systems are evaluated.

The purpose of these simulations was to determine whether a controller developed using the linearized model,

will produce a stable response of the nonlinear model under investigation. If the response is stable, the gains calculated for the linearized deterministic system are suitable to be initialized in the adaptive controller of the nonlinear system.

Both systems were given the same one inch step input and same position control law that was derived in the last section.

$$I = [K] \{ r(t) - X(t) \} , \quad (3.3-1)$$

where  $\{r\}$  - the input vector

$\{X\}$  - the state vector

and  $[K] = \begin{bmatrix} 502.228 & 10.4174 & .1295058 & -.1295058 \end{bmatrix}$

is the gain matrix.

Figure 10 shows the simulated response of the linearized model and Figure 11 shows the simulated response of the nonlinear model. The initial conditions were identical for the two simulations. The actuator position converged in both simulations to one inch (the same as the step input) and the control variable converged to zero. The other states recorded,  $P_1$  and  $P_2$ , are far from being identical due to the differences between the linearized model and the nonlinear model, but a stable response is shown for both models.

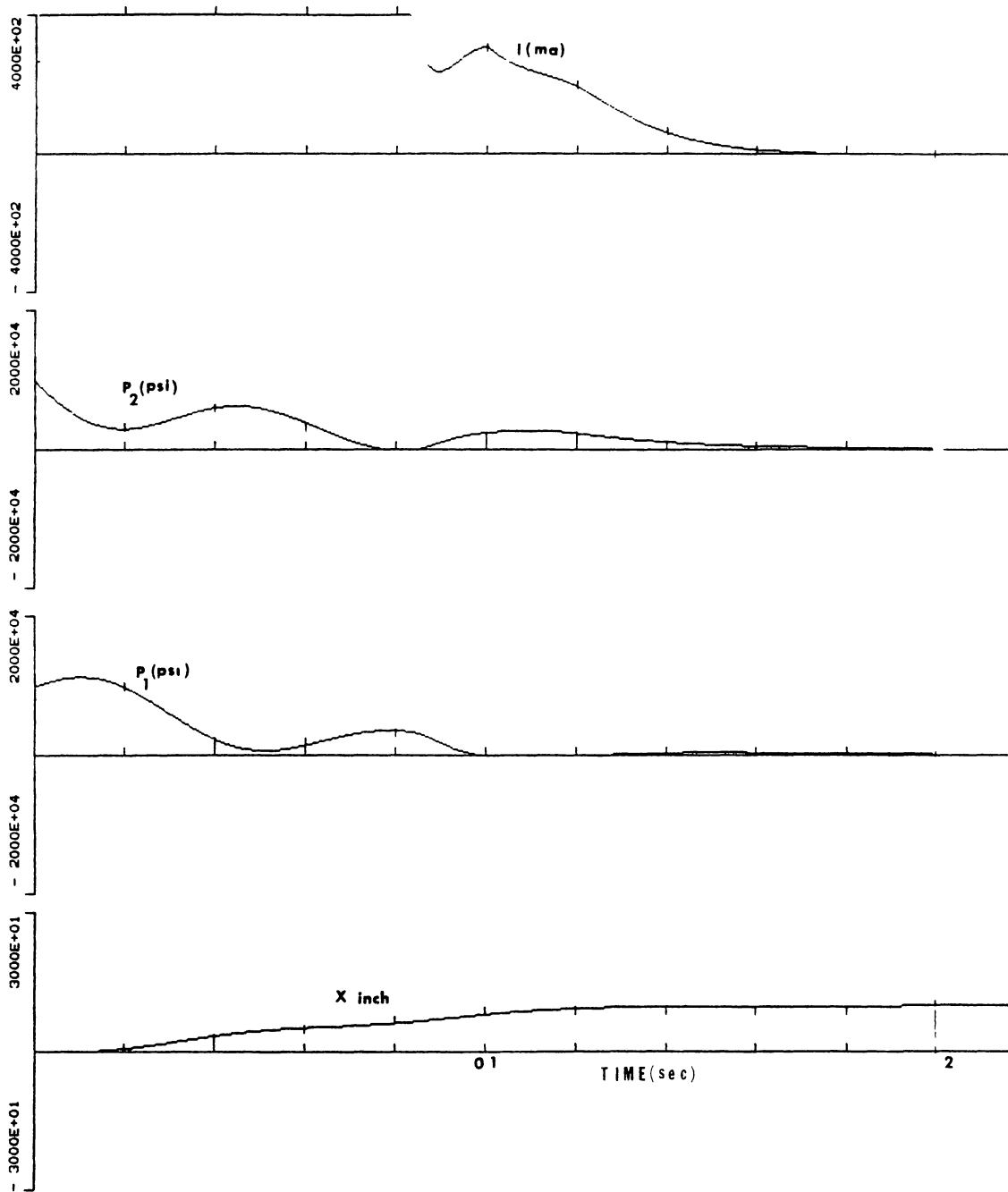


Figure 10. Simulated Response of Linearized model  
to 1" step input.  
(Deterministic System, constant gains)



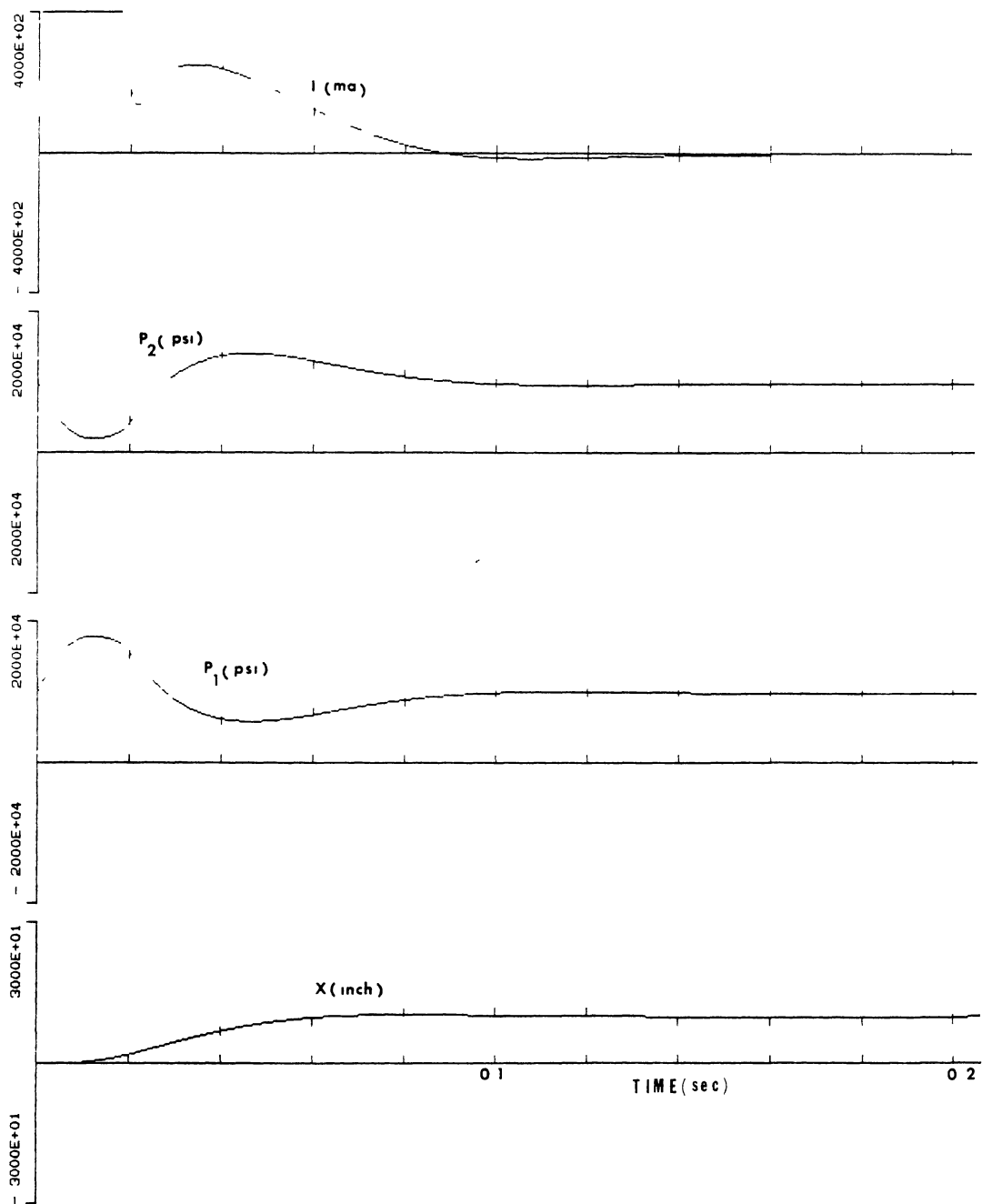


Figure 11. Simulate Response of Nonlinear model  
to 1" step input.  
(Deterministic System, constant gains)

### 3.4 Adaptive Control Law Simulation and Discussion

#### 3.4.1 Simulation Program

A computer program was developed to simulate the identification, control synthesis and washout procedures discussed in Chapter II. The application model and proposed control system were described in section 3.0 of this chapter.

3.4.1.1 Program Description. The program includes a main calling program and eight subroutines. This program was written in the structured programming language Fortran 77. A flow chart description of the program is given in Figure 12.

3.4.1.2 Program NLCONT. This is the main program used to initialize parameters, to control timing and to store interim results. The program calls the integration routine (RK41) to simulate the "plant" response and calls the optimization routine (QNDAV1) twice: (1) for parameters identification and (2) for gain coefficients calculations.

3.4.1.3 Subroutine RK41. The purpose of the subroutine is to generate the input signal to the system, calculate the control variable and integrate the nonlinear system equations of motion. The input is generated for two modes: the follower mode and the washout mode. Integration is done

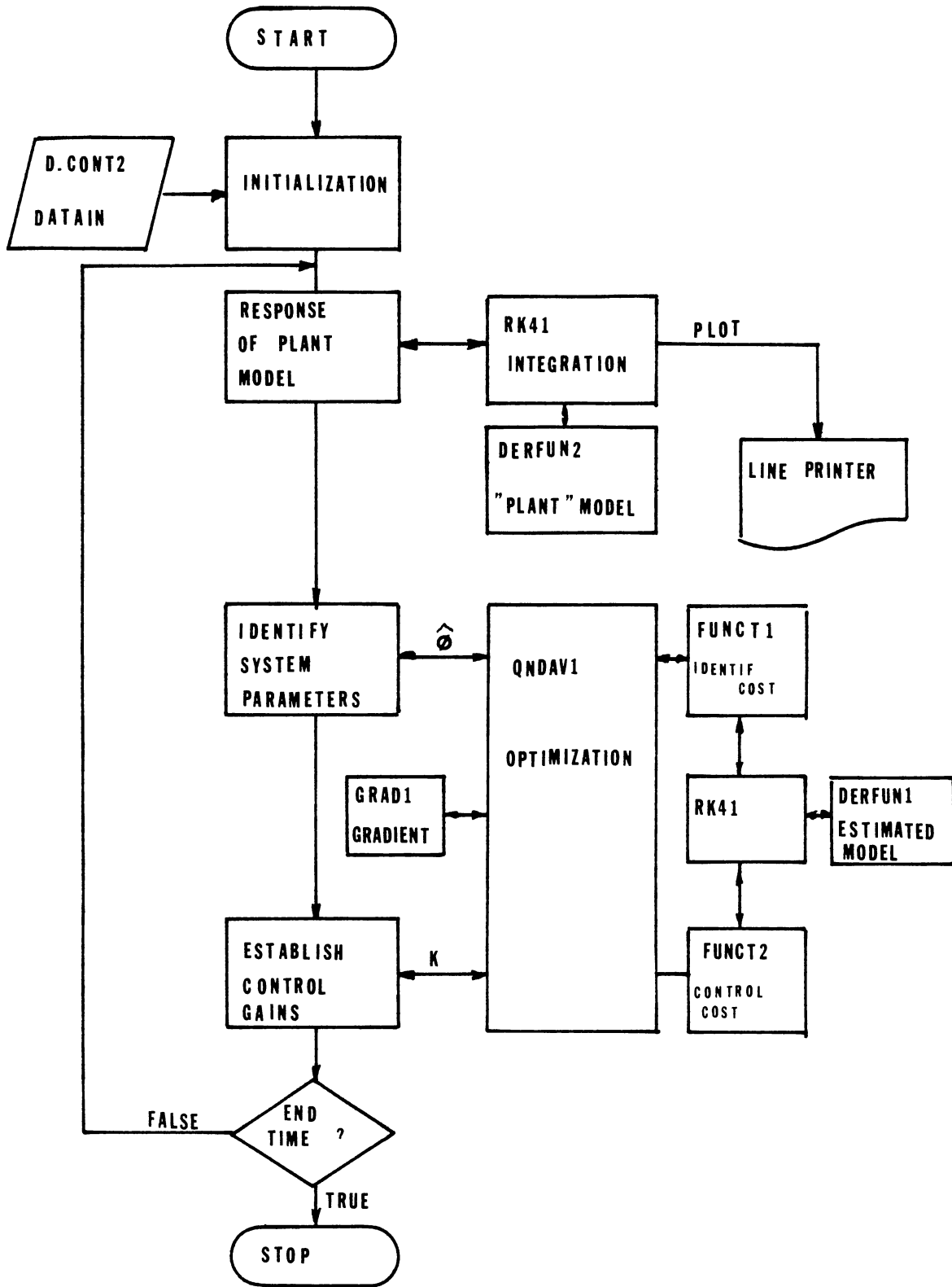


Figure 12. Simulation Program Flowchart.

using a fourth-order Runge-Kutta method. This subroutine is called by the main program (NLCONT) and by the cost function subroutines (FUNCT1, FUNCT2). The subroutines called by RK41 are the system equations of motion: DERFUN1 - the estimated model; DERFUN2 - the reference plant model.

3.4.1.4 Subroutine QNDAV1. The purpose of this subroutine is to calculate the minimum of a nonlinear function of N variables and to establish the values of the variables for the minimum point.

This program was written to implement the algorithm suggested by W.C.Davidon (25), which is basically a quasi-Newton method without line search. The subroutines called by QNDAV1 are: FUNCT1 (identification cost function), FUNCT2 (control cost function) and GRAD1 (gradient subroutine).

3.4.1.5 Subroutine GRAD1. The purpose of this subroutine is to calculate the gradient of a function of N variables. The subroutines called by GRAD1 are: FUNCT1 (identification cost function) and FUNCT2 (control cost function).

3.4.1.6 Subroutine FUNCT1. The purpose of this subroutine is to calculate the cost function in the parameter identification process. The subroutine called by FUNCT1 is the integration subroutine RK41.

3.4.1.7 Subroutine FUNCT2. The purpose of this subroutine is to calculate the cost function for the control synthesis process. The subroutine called by FUNCT2 is the integration subroutine RK41.

3.4.1.8 Subroutine DERFUN1. The purpose of this subroutine is to calculate the equations of motion of the estimated model.

3.4.1.9 Subroutine DERFUN2. The purpose of this subroutine is to calculate the equations of motion of the reference plant model.

3.4.1.10 Subroutine LXYPLT. The purpose of this subroutine is to generate plots on the Printronix line printer (at Burtek, Inc., Tulsa, OK.).

### 3.4.2 Simulation Results for the Adaptive Control System

In this section, the simulated responses of the adaptive control system are discussed and compared with the dynamic characteristics of an existing motion platform.

An acceleration input was chosen for the simulation, since it will transfer realistic cues to the pilot trainee. If a position input were selected, the trainee would get a cue which lags the acceleration of the aircraft by 180°.

Two parameters were selected for identification: Servo valve gain and bulk modulus of the hydraulic fluid. The "plant" parameters were fixed as follows:

$$\text{Valve gain} = 0.172 \text{ [in}^4\text{/sec-ma-Lbf}^{\frac{1}{2}}\text{]}$$

$$\text{Bulk modulus} = 0.15 \text{ [x10}^6\text{Lbf/in}^2\text{]}.$$

The values for the estimated model parameters were initialized for all runs as follows:

$$\text{Valve gain} = 0.180$$

$$\text{Bulk modulus} = 0.140$$

The nonlinear system that was described in section 3.2 was tested by the adaptive control simulation program of section 3.4.1 for three different time-dependent signal inputs:

$$\begin{aligned} r_1 &= 22.0 \sin 3t && \text{[in/sec}^2\text{]} \\ r_2 &= 120.0 \sin 10t && \text{[in/sec}^2\text{]} \\ r_3 &= 80.0t && \text{[in/sec}^2\text{]}. \end{aligned}$$

The system responses for the above inputs are given respectively in Figure 13, Figure 14 and Figure 15.

The low frequency input (Figure 13) and high frequency input (Figure 14) were selected from a typical simulator operating envelope chart (Figure 7).

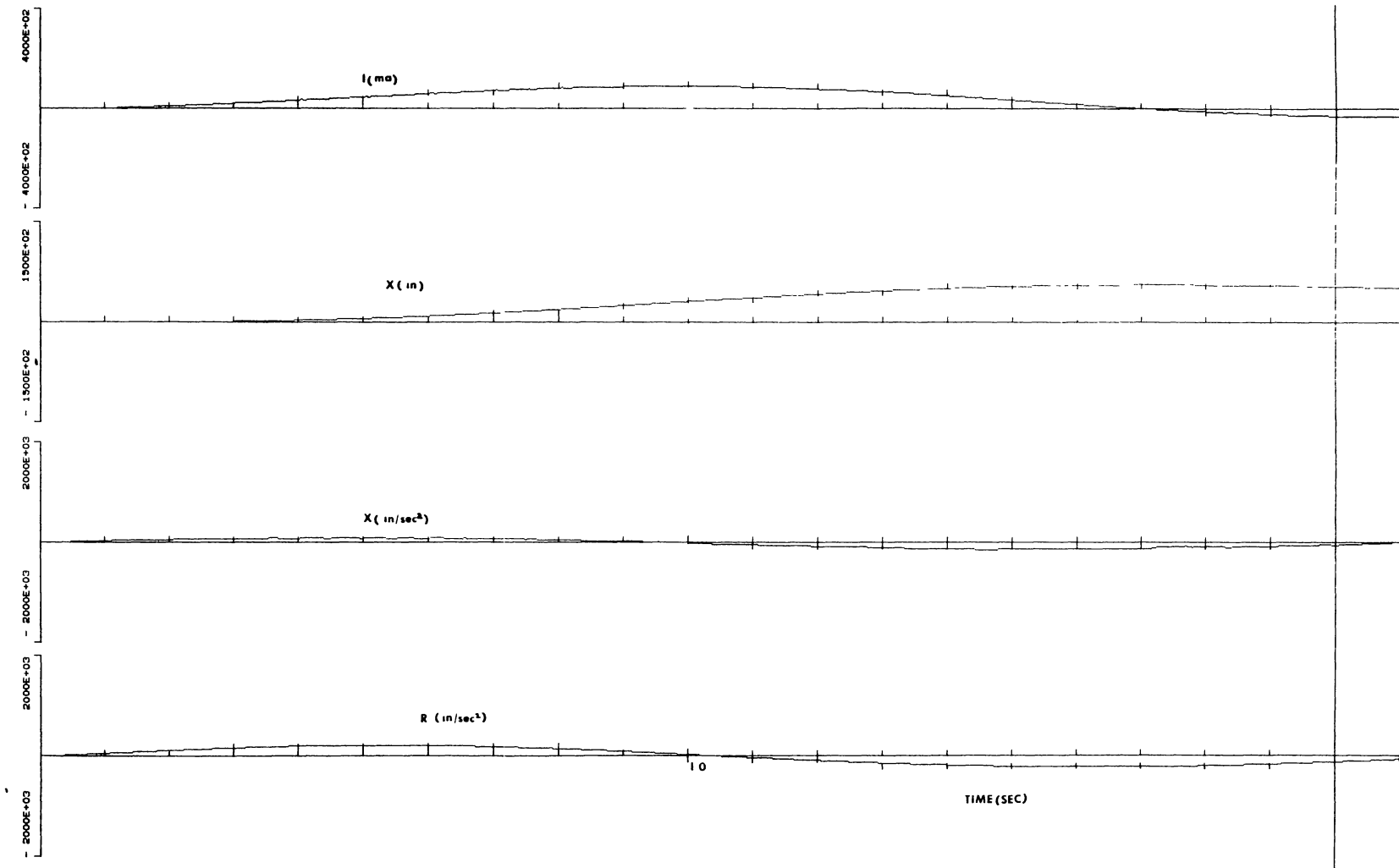


Figure 13. Low Frequency Input  
(Adaptive Control System).

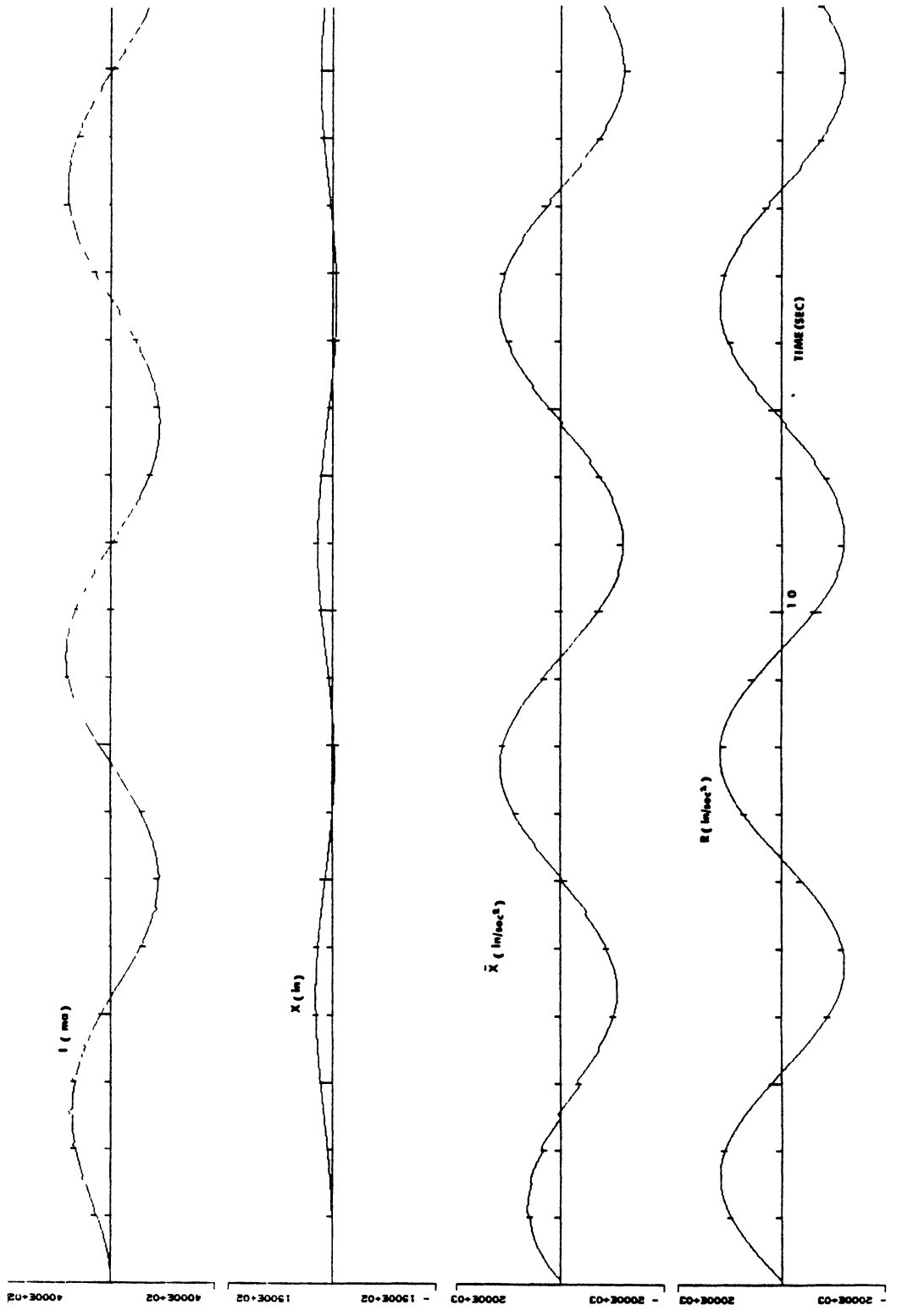


Figure 14. High Frequency Input  
(Adaptive Control System).



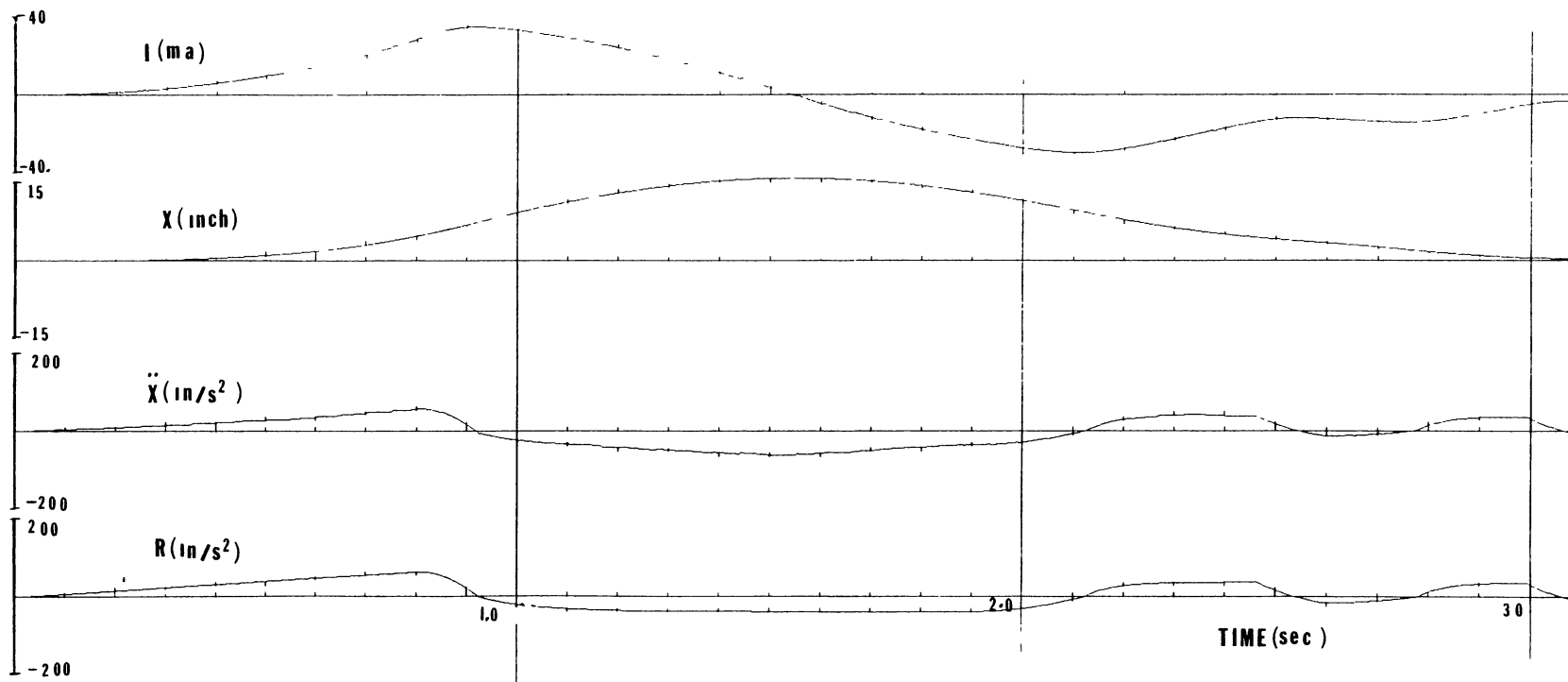


Figure 15. Ramp Input and Washout Process

The ramp input (Figure 15) was selected to test the washout algorithm. Figure 13 shows a good follow-up response of the system to a low-frequency input. But, because of the small amplitude of the input and the outputs, it is hard to make any decisive evaluation of the response.

The results in Figure 14 show that for a high frequency input, the phase lag and attenuation recorded became negligible after a short time interval. This short time interval (about one second) was needed to adapt the parameters of the model to the system parameters and to adjust the gains. The updates in the gains are noticed as minor discontinuities in the control  $I$  and in the response  $X$  during the initial time interval discussed above.

Figure 15 shows the response of the system to a ramp input. After about 0.8 sec it was calculated that the actuator limit may be exceeded, and the mode of operation was switched to the washout mode. This mode is intended to bring the motion platform to a neutral position ( $X=0$ ), with an acceleration ( $\ddot{X}$ ) limit of 0.1g. During the movement of the platform towards the neutral position, the input to the system is adjusted continuously, so that the platform will not overshoot the neutral position. Figure 16 shows the estimated values of the identified parameters as a function of time as the simulation progresses. The system input was  $120 \sin 10t$  [in/sec<sup>2</sup>] as shown in Figure 14. The parameters estimated were: valve gain (nominal value = 0.172) and bulk

modulus of the fluid (nominal value =  $0.150 \times 10^6$ ).

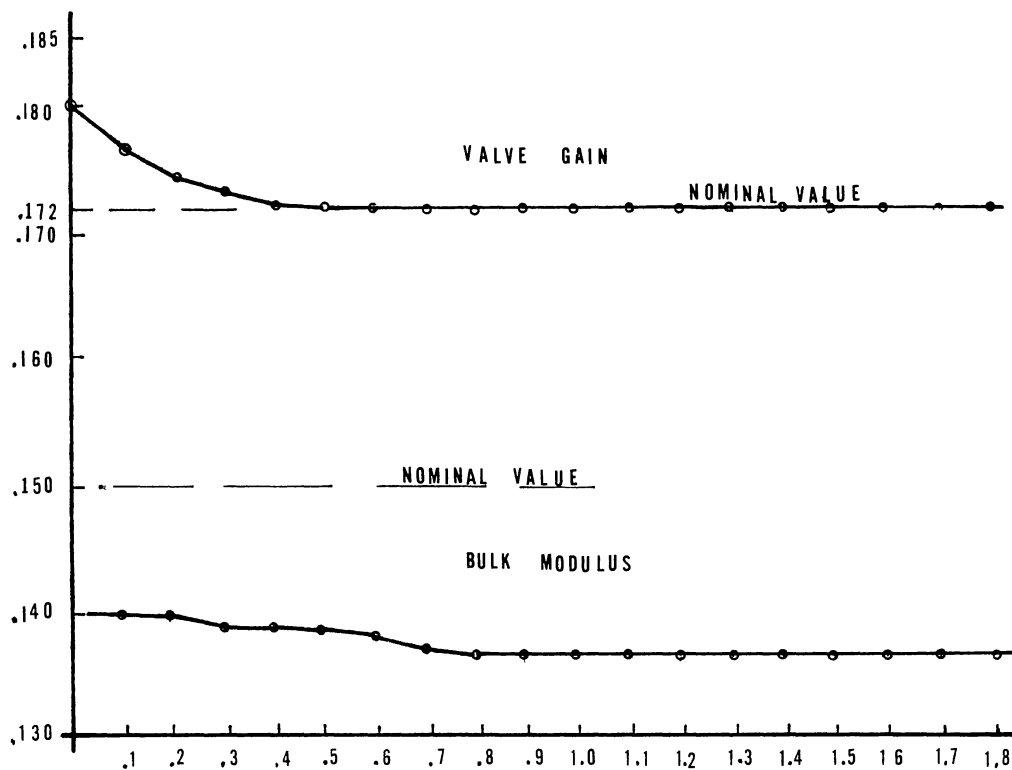


Figure 16. Parameter Estimation During Run Time

(Input  $r=120 \sin 10t$  [in/sec<sup>2</sup>])

$$Q = [1.0, 1.0 \times 10^{-2}, 1.0 \times 10^{-4}, 1.0 \times 10^{-4}]$$

The valve gain converged to the nominal value at about 0.5 sec. The estimated bulk modulus of the fluid did not converge to the nominal value, but it reached a constant value after about 0.7 seconds. This result indicates that the system performance is not very sensitive to the value of

the bulk modulus except near resonance. (The estimated bulk modulus of the fluid converged to the nominal value in one of the simulation runs, when instability was introduced into the system).

Figure 17 shows the adjustment of the gains as they occurred during the computation for the high frequency input ( $120 \sin 10t$ ).

These adjustments occur automatically during the initial transient and settle to a nearly constant value after 1.5 seconds. The mode selected was acceleration (pressure) control and the initial gains selected were arbitrary. If the initial gains were selected by calculating the nominal gains from the linearized model, the convergence of the computed gains would be accelerated.

The gains displayed in Figure 17 were calculated for a particular input. Thus, different inputs lead to different gains.

Since identification and control synthesis depend upon the ability to minimize a defined cost function, a quantitative evaluation of the method can be made by investigating the values of the cost function during the iteration process. Table I gives the identification cost functions calculated for each iteration. The identification cost function is defined in section 2.2. The control cost function as defined in section 2.1 is given for each iteration in Table II.

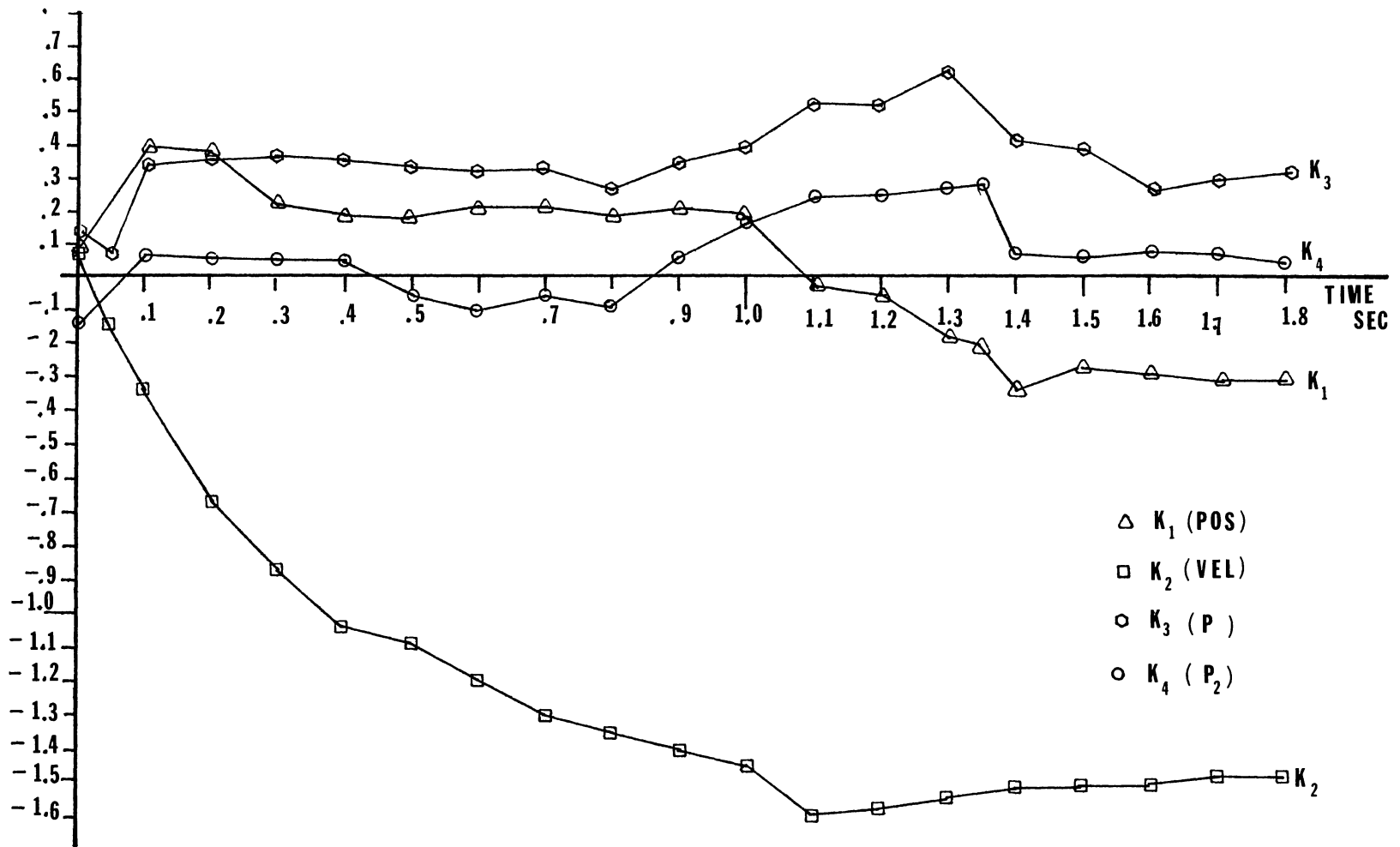


Figure 17. Gain Adjustment During Run Time  
(Input.  $120 \sin 10t$ )

TABLE I  
 RESULTANT ERROR FUNCTION OF IDENTIFICATION ALGORITHM.  
 (at 0.05 sec. update)

Iteration No.	Function Value
1	.89 x 10 <sup>-4</sup>
2	.11 x 10 <sup>-3</sup>
3	.979 x 10 <sup>-7</sup>
4	.349 x 10 <sup>-7</sup>
5	.376 x 10 <sup>-7</sup>
6	.357 x 10 <sup>-7</sup>

Both tables are given for simulation results shown in Figure 14. Both accepted points and rejected iteration points were recorded in these tables. Table III is a recording of the number of function evaluations and gradient calculations performed during the system simulation. These values are used to predict computation time needed for real time execution (section 4.2).

The results of the above adaptive control law simulation can be used to qualitatively evaluate the method against a traditional design of similar systems.

TABLE II  
 RESULTANT ERROR FUNCTION OF CONTROL ALGORITHM  
 (at 0.05 sec. update)

Iteration No.	Function Value
1	$.131 \times 10^4$
2	$.127 \times 10^5$
3	$.305 \times 10^3$
4	$.309 \times 10^7$
5	$.265 \times 10^4$
6	$.668 \times 10^2$
7	$.523 \times 10^3$
8	$.849 \times 10^1$
9	$.278 \times 10^1$
10	$.469 \times 10^0$
11	$.217 \times 10^{-1}$
12	$.129 \times 10^{-1}$
13	$.109 \times 10^{-1}$

Figure 18 and 19 show experimental results for small amplitude sinusoidal position inputs to an existing motion platform actuator, employing constant feedback gains.

For an input frequency of 10 rad/sec, the phase lag of the system with position, velocity and differential pressure feedback (Figure 18) is above  $55^\circ$ , and the attenuation is about -1.5 dB. For the system with only position and dif-

TABLE III

NUMBER OF FUNCTION AND GRADIENT EVALUATIONS DURING SIMULATION (6 function calls allowed for identification and 13 for control optimization).

Update Time (sec)	Identification		Control	
	Function	Gradient	Function	Gradient
0.05	6	3	13	9
0.10	6	3	13	5
0.15	6	2	13	4
0.20	6	3	13	4
0.25	6	3	13	5
0.30	6	3	13	5
0.35	6	3	13	5
0.40	6	3	13	3
0.45	6	3	13	3
0.50	6	3	13	4
0.55	6	4	13	5
0.60	6	4	13	3
0.65	6	5	13	4
0.70	6	5	13	3
0.75	6	2	13	5
0.80	6	4	13	3
0.85	6	3	13	4
0.90	6	4	13	3
0.95	6	1	13	2
1.00	6	1	13	4



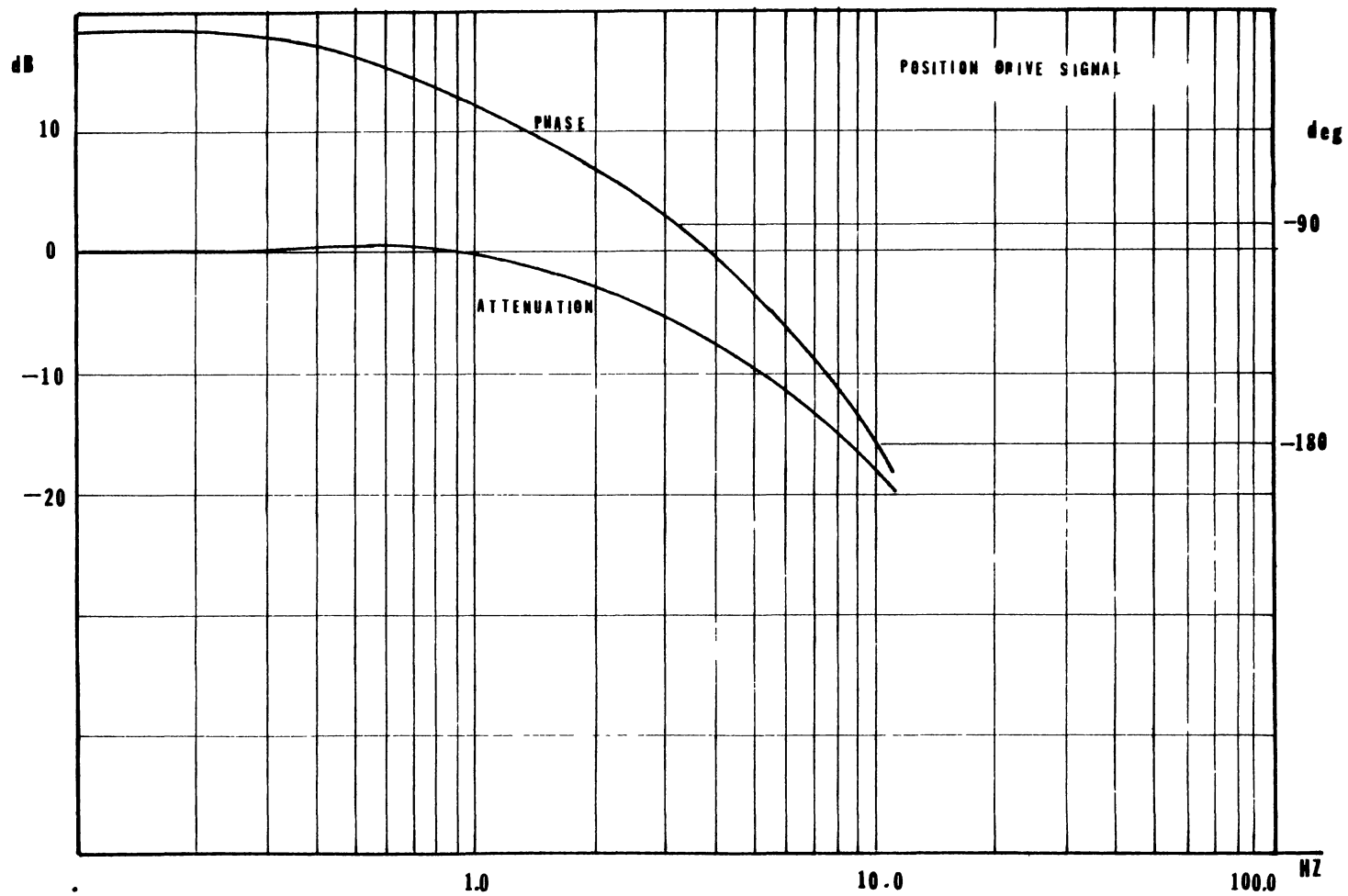


Figure 18. Frequency Response in Heave for Existing System (With Pressure and Velocity Feedback)

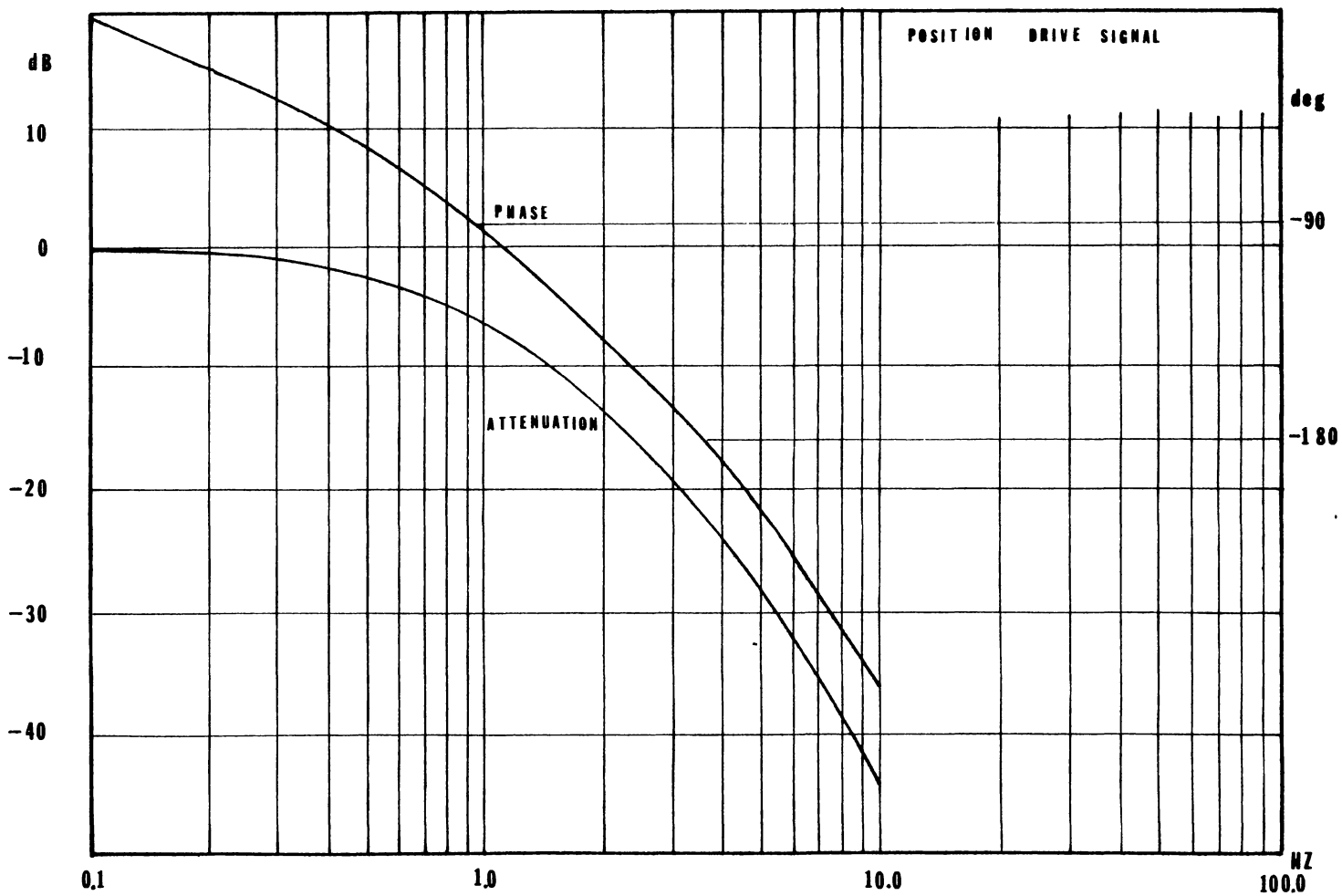


Figure 19. Frequency Response in Heave for Existing System (Pressure Feedback, no Velocity Feedback).

ferential pressure feedback (Figure 19), the phase lag (the same input) is  $128^\circ$  and the attenuation is  $-5.0$  dB. The experimental results will give other characteristics for larger amplitude inputs since the system is highly nonlinear.

The experimental frequency response curves of Figure 18 and 19 show degradation in performance for high frequency inputs. By comparison, the results of the adaptive control law simulation of Figure 14 (time domain response) shows no degradation of the system for  $10$  rad/sec frequency input; both attenuation and phase lag are negligible.

In Figure 20, the conventional method of adjusting feedback gain values is depicted. The system is provided with a position step input and the actuator differential pressure output is recorded as a function of time. The pressure feedback gain (which is dependent on a resistance in the gain amplifier) is adjusted until the differential pressure recorded shows fast decay with an acceptable level of overshoot and no oscillations. This is a lengthy way to get the correct gain values. Whenever more than one feedback gain needs to be adjusted, the number of tests needed in order to adjust the system gains increase. These gains will be satisfactory only for a limited envelope and need to be rechecked and adjusted periodically because of drift. On the other hand, using the method devised in this thesis, the system gains are adjusted continuously by

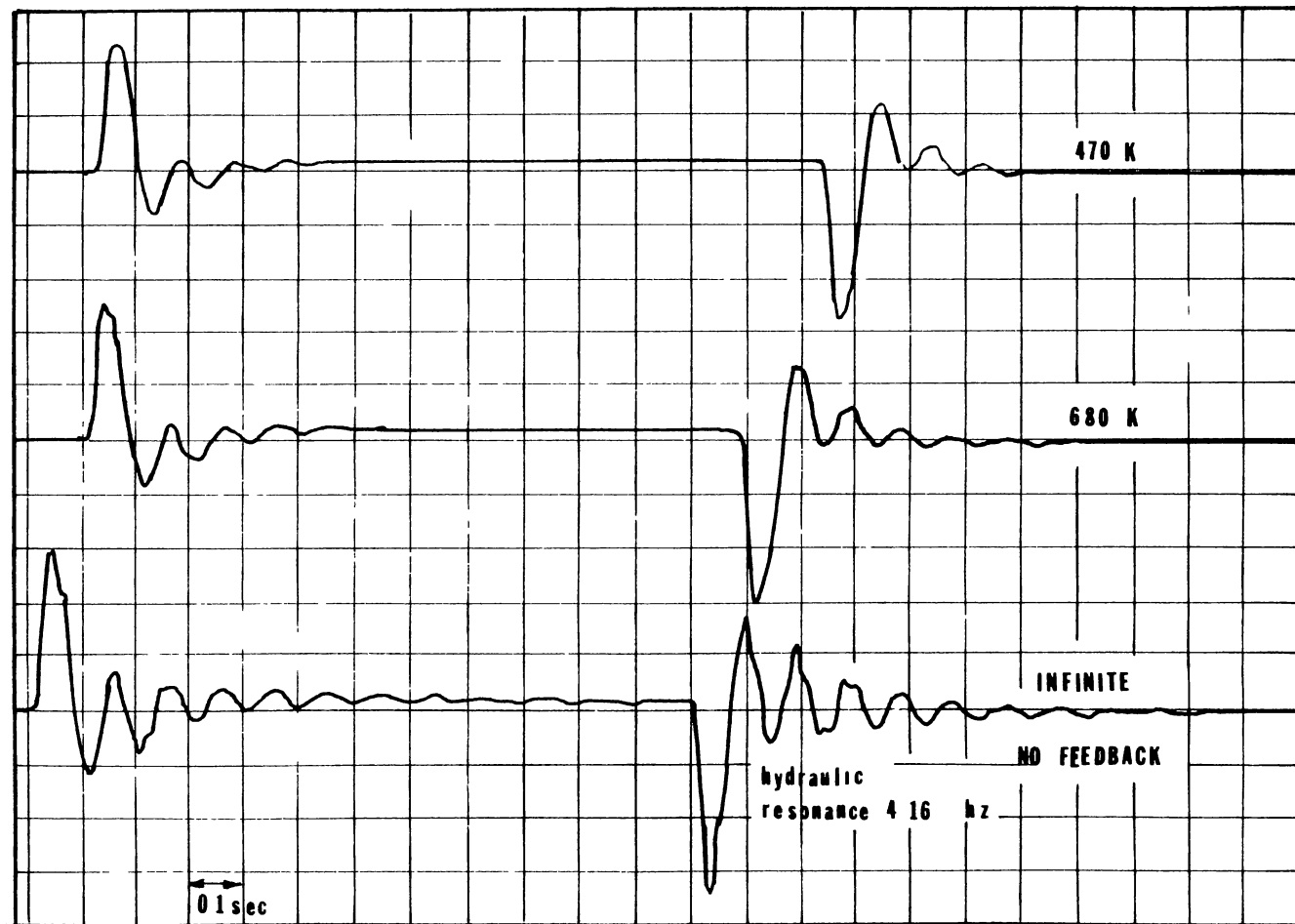


Figure 20. Measured Differential Pressure vs. Time  
for Different Pressure Feedback Gains.

the algorithm, thereby taking care of the nonlinearities and the drifts in system parameters.

## CHAPTER IV

### IMPLEMENTATION CONSIDERATIONS

#### 4.0 GENERAL

Chapter III was concerned with the application of the proposed method to on-line control of a one degree-of-freedom motion platform. The control law was formulated, and the model was derived and coded in Fortran Language. Some tests were conducted and the system response was simulated using off-line aids like a nominal system model and plots. By using the trajectories obtained from the simulation runs and comparisons with existing control methods, controller performance was evaluated. The following sections are devoted to real-time implementation feasibility studies, especially the memory requirements, run-time prediction and expected noise.

#### 4.1 Memory Requirements

Table IV shows the memory requirement for off-line program execution. These object code requirements include both program and data storage. The total memory requirement for off-line simulation of programs and data (FORT77 Compiler) on the SEL computer is 48K bytes.

Davidon (25) reported that the nonlinear optimization routine as programmed on a Wang 720 mini-computer, used less than 672 bytes of memory for program storage. Since the object code created by the SEL FORTRAN-77 Compiler was 9072 bytes, then even if the data memory (20 parameter capability, 5880 bytes) requirement is subtracted, there is still a need for 3192 bytes of program memory. It is evident that for real time execution, the code has to be changed, optimized and written in assembler language.

TABLE IV  
MEMORY REQUIREMENTS FOR OFF LINE PROGRAM EXECUTION

Program	Function	Object Code (Bytes)
NLCONT	Main	1592
LXYPLT	Plot graphs	3576
QNDAV1	Nonlinear function minimization	9072*
RK41	Integration	1048
FUNCT1	Identification cost function	176
FUNCT2	Control cost function	176
GRAD1	Numerical gradient	272
DERFUN1	'Plant' model	536
DERFUN2	'Update' model	536

\* QNDAV1 can handle up to 20 parameters

Using Davidon's data for the storage requirement for the minimization routine, it can be expected that for real-time execution, the storage requirement for the application discussed in this thesis is about 5K bytes of memory. This storage requirement is very modest with respect to the capacity of most modern mini or micro computers.

#### 4.2 On-Line Execution Time

The on-line execution time for the algorithm can be determined by counting the total number of additions, multiplications, function evaluations and gradient evaluations for a given iteration of the minimization routine and multiplying the operation count by a representative execution time for these operations. The communication time between the processor and the memory is ignored. This estimate, however, is problem dependent.

The dominant time consumer in the minimization routine is the function evaluation time which is called by the minimization subroutine and by the gradient subroutine.

A function evaluation consists of integrating a set of differential equations over a defined time interval and computing a suitably defined error function. Thus, the function evaluation time depends primarily on the numerical integration method employed and the complexity of the differential equations being employed.



#### 4.2.1 Number of Operations Per Integration Step

Because of its extensive use in a wide variety of areas, as well as its high degree of accuracy, the fourth-order Runge-Kutta (RK4) integration method was used exclusively to obtain the results presented in the previous sections. For the run time estimation, three other integration methods were chosen for comparison: second-order Runge-Kutta method (RK2), Adams-Bashforth second-order predictor (AB2) and Adams-Moulton (Predictor-corrector) fourth-order (AM4). A short review of these methods and the selection of the integration step size are given in Appendix B.

The following letters may be assigned to represent the number of arithmetic operations:

n = the dimensionality of the system

A = additions/subtractions

M = multiplications/divisions

F = function evaluations.

For RK2 implementation, the total arithmetic operations which must be carried out per step:

$$2F + (4A + 2M)n \qquad (4.2.1-1)$$

For RK4 implementation the number is

$$4F + (9A + 11M)n \quad (4.2.1-2)$$

For the AB2 predictor the number is

$$1F + (2A + 3M)n \quad (4.2.1-3)$$

and for the AM4 predictor the number is

$$2F + (8A + 11M)n \quad (4.2.1-4)$$

Number of arithmetic operations for evaluating  $\dot{x} = f(x)$  for the example considered:

$$20M + 8A + 2 \text{ square root evaluations}$$

Each iteration of a square root can be regarded as  $4M + 2A$  evaluations. Considering only one square root iteration per call, the total number of operations will be

$$F = 20M + 8A + (4M + 2A) = 24M + 10A \quad (4.2.1-5)$$

The number of operations required for a single integration step is given in Table V for  $n = 4$ .

TABLE V  
 NUMBER OF ARITHMETIC OPERATIONS NEEDED  
 PER SINGLE INTEGRATION STEP

Method	Additions	Multiplications	Weighted operations assuming $t_m = 2t_A$
RK4	76	140	356
AM4	52	92	236
RK2	36	56	148
AB2	18	36	90

As reported earlier, the off-line tests of the proposed algorithm employed the RK4 method because of its simplicity in coding and its self start characteristics and not because of its efficiency (see Table V). The Adams-Moulton formulas have a significantly smaller truncation error than the Adams-Bashforth formulas, for comparable order methods. For example, the fourth-order Adams-Moulton formula has a truncation error 0.076 times that of the fourth-order Adams-Bashforth formula. This is the principal reason for using the implicit formulas, although there are other considerations.

The fourth-order Adams-Moulton formula has over twice the truncation error of the Simpson method

$$(X_{i+1} = X_{i-1} + \frac{h}{3}[f(X_{i-1}, t_{i-1}) + 4f(X_i, t_i) + f(X_{i+1}, t_{i+1})]).$$

The reason for using the Adams-Moulton formula is that it has much better stability properties than the Simpson rule.

Based on the above results and considerations, it is concluded that for real time applications the integration method used should be the AM4 or the AB2, and not the RK4 as implemented for off-line studies.

#### 4.2.2 Number of Arithmetic Operations per Function Evaluation

Let L = number of steps of the integrations

I = number of operations per integration

P = number of parameters to be identified

Then the number (R) of operations per function evaluation will be:

$$R = PL(3A + 2M) + LI \quad (4.2.2-1)$$

Substituting some representative numbers and using the AM4 integration method:

$$P = 4 \text{ for control calculations}$$

$$I = 52A + 92M$$

$$L = 10$$

Then

$$\begin{aligned} R &= 4 \times 10(3A + 2M) + 10(52A + 92M) = 640A + 1000M \\ &= 2640 \text{ equivalent operations} \end{aligned}$$

For the AB2 integration method:

$$P = 4$$

$$I = 18A + 36M$$

$$L = 5$$

$$\begin{aligned} R &= 4 \times 5(3A + 2M) + 5(18A + 36M) = 150A + 220M \\ &= 590 \text{ equivalent operations} \end{aligned}$$

#### 4.2.3 Number of Arithmetic Operations per Gradient Evaluation

Let  $G$  = number of operations needed for gradient evaluation

$$G = P.R \quad (4.2.3-1)$$

Tables VI and VII summarize the number of operations needed for function and gradient evaluation, assuming  $t_m = 2t_A$ .

TABLE VI  
 NUMBER OF WEIGHTED OPERATIONS NEEDED FOR  
 FUNCTION AND GRADIENT EVALUATION  
 (5 steps of integration)

phase	Integration Method	
	AM4	AB2
Identification	Function: 1250	Function: 520
P = 2	Gradient: 2500	Gradient: 1040
Control	Function: 1320	Function: 590
P = 4	Gradient: 5280	Gradient: 2360

TABLE VII  
 NUMBER OF WEIGHTED OPERATIONS NEEDED FOR  
 FUNCTION AND GRADIENT EVALUATION  
 (10 steps of integration)

phase	Integration Method	
	AM4	AB2
Identification	Function: 2500	Function: 1040
P = 2	Gradient: 5000	Gradient: 2080
Control	Function: 2640	Function: 1180
P = 4	Gradient: 10560	Gradient: 4270

#### 4.2.4 Minimization Program - Number of Arithmetic Operations

The number of arithmetic operations for one iteration of the minimization routine (excluding function and gradient evaluation) are arranged in the order of the steps given in Appendix A:

$$\text{Step 1: } P ( 1A + 4M ) + 1A$$

$$\text{Step 2: } P^2( 1A + 1M ) + PM + 1M$$

$$\text{Step 3: } P^2( 1A + 1M ) + P(2M + 3A) + 1M$$

$$\text{Step 4: } P ( 8A + 11M ) + 7M + 1A$$

$$\text{Step 5: } P ( 2A + 4M ) + A + 2M$$

$$\text{Step 6: } 56M + 26A$$

$$\text{Step 7: } P^2( 2M + 1A ) + P(20M + 10A) + 3A + 3M$$

The total number of operations (less function and gradient evaluation) needed per minimization iteration is given by

$$O = P^2(3A+4M) + P(24A+42M) + 32A + 70M \quad (4.2.4-1)$$

using equation (4.2.4-1), the total operations for identification ( $p=2$ ) is

$$O_1 = 92A + 170M = 432 \text{ weighted operations}$$

and the total operations for control evaluation ( $P=4$ ) is

$$O_2 = 176A + 302M = 780 \text{ weighted operations}$$

Assuming 6 calls for function and 3 calls for gradient evaluation during the identification process (from Table III), the number of operations using AM4 (10 integration steps) is given by

$$N_1 = 6 \times 2500 + 3 \times 5000 + 432 = 30,432$$

and the number of operations using AB2 integration method is

$$N_1 = 6 \times 1040 + 3 \times 2080 + 432 = 12,912$$

Assuming 13 calls for function and 5 calls for gradient evaluation during the control calculation process (from Table III), the number of operations using AM4 (10 integration steps) is

$$N_2 = 13 \times 2640 + 5 \times 10560 + 780 = 87,120$$

and using the AB2 integration method

$$N_2 = 13 \times 1180 + 5 \times 4720 + 780 = 39,720$$

is obtained. Using the above values for  $N_1$  and  $N_2$ , the total number of operations using the AM4 method is



$$N_1 + N_2 = 30432 + 87120 = 117,552$$

and the total number of operations using the AB2 method is

$$N_1 + N_2 = 12912 + 39720 = 52632$$

Taking the instruction time as  $0.30 \mu\text{sec}$  (Z-8002 micro-processor), one cycle of the combined identification and control process will require an estimated execution time of

$$T = 117552 \times .3 \times 10^{-6} = 0.0353 \text{ sec. (AM4), or}$$

$$T = 52632 \times .3 \times 10^{-6} = 0.0158 \text{ sec. (AB2).}$$

Since one iteration of the adaptive control optimization (both identification and control) consumes about 0.035 sec. (AM4) or 0.016 sec. (AB2), it appears that for the application in this thesis, serial processing will be adequate. The step size for the integration routine does not play much of a role in the computational time, but it should be truncation error dependent, as explained in Appendix B. The truncation error limit should be no less than the order of the measurement resolution in order to conserve evaluation time.

#### 4.2.5 Prediction of Timing

The motion platform hardware exhibits a natural frequency of 4 HZ (Fig. 20). The correct values for the integration step size should be determined by off-line simulation employing a variable step method that depends upon the truncation error. Assuming here that integration step size is 1/25 of the natural period of the system to be controlled:

$$h < \frac{1}{4} \times \frac{1}{25} = 0.010 \text{ sec.}$$

The maximum frequency of input to the system (from aircraft dynamics) is normally 2 HZ. Assuming an update of the control law at a rate of  $2 \times 6 = 12$  HZ, then the update time is  $1/12 = 0.080$  sec.

This update time gives enough processor time to execute the proposed algorithm with more than 50% spare time when the AM4 integration method is used, or more than 80% when the AB2 integration method is used. Since the integration is executed during a fraction of the update time, the step size may be reduced, and a less accurate but faster integration method like the AB2 could be used.

### 4.3 Noise Expectation

A real nonlinear dynamic system and measurement model may be represented by

$$\dot{x}(t) = f[x(t), t] + B(t)w(t) \quad (4.3-1)$$

$$z(t) = h[x(t), t] + v(t) \quad (4.3-2)$$

where  $w(t)$  is an  $m$ -vector of process noise and  $v(t)$  is an  $r$ -vector of measurement noise that corrupts the observation  $z(t)$ . Few approaches are given in the literature for real-time estimation where the noise processes  $w(t)$  and  $v(t)$  are mutually independent zero-mean white noise processes. Kaufman and Teavassos (20) suggested solving the above problem using an optimal estimation method.

A feasibility study for real-time utilization of a continuous Kalman filter was given by Gaston and Rowland (17). But, this method relies on the linearization of the model around operating point and not directly on the nonlinear equations of motion.

Sinaha and Kuszta (27) presented the regression method used to select the parameters of the model  $Y_k = f(k) + W_k$ , such that  $W_k$  is a zero-mean white noise sequence of least-possible variance. This latter method involves determining the parameters that will minimize the mean-square error

$$J = \frac{1}{N} \sum_{k=1}^N (Y_k - f(k))^2 = \frac{1}{N} \sum_{k=1}^N W_k^2$$

This method is comparable to the direct estimation method used in this thesis. Kauffman and Teavassos recommended the

use of optimal estimation method in the presence of appreciable noise but did not recommend limits for the validity of the direct estimation method. It appears that the reason for not recommending limits is that these limits are problem dependent and the approach taken should be left at the discretion of the system designer.

In order to predict whether noise might constitute a problem for the application under study, an experiment was conducted using existing simulator hardware. A position step input command was given to one of the three actuators controlling the C141 flight simulator motion platform and the actuator pressure transient was recorded using a Gould 2400S thermal writing recorder. This test was repeated several times and the response plots for the various tests are compared in order to determine the presence of independent noise input to the system.

Figure 21 is a diagram of the pressure measurement circuits. Pressure transducers (sensitivity of 200mv/3000 psi) mounting in actuator chambers 1 and 2, transmit the measurements via amplifiers (gain value of about 50). The summer amplifier then gives the differential pressure (Test

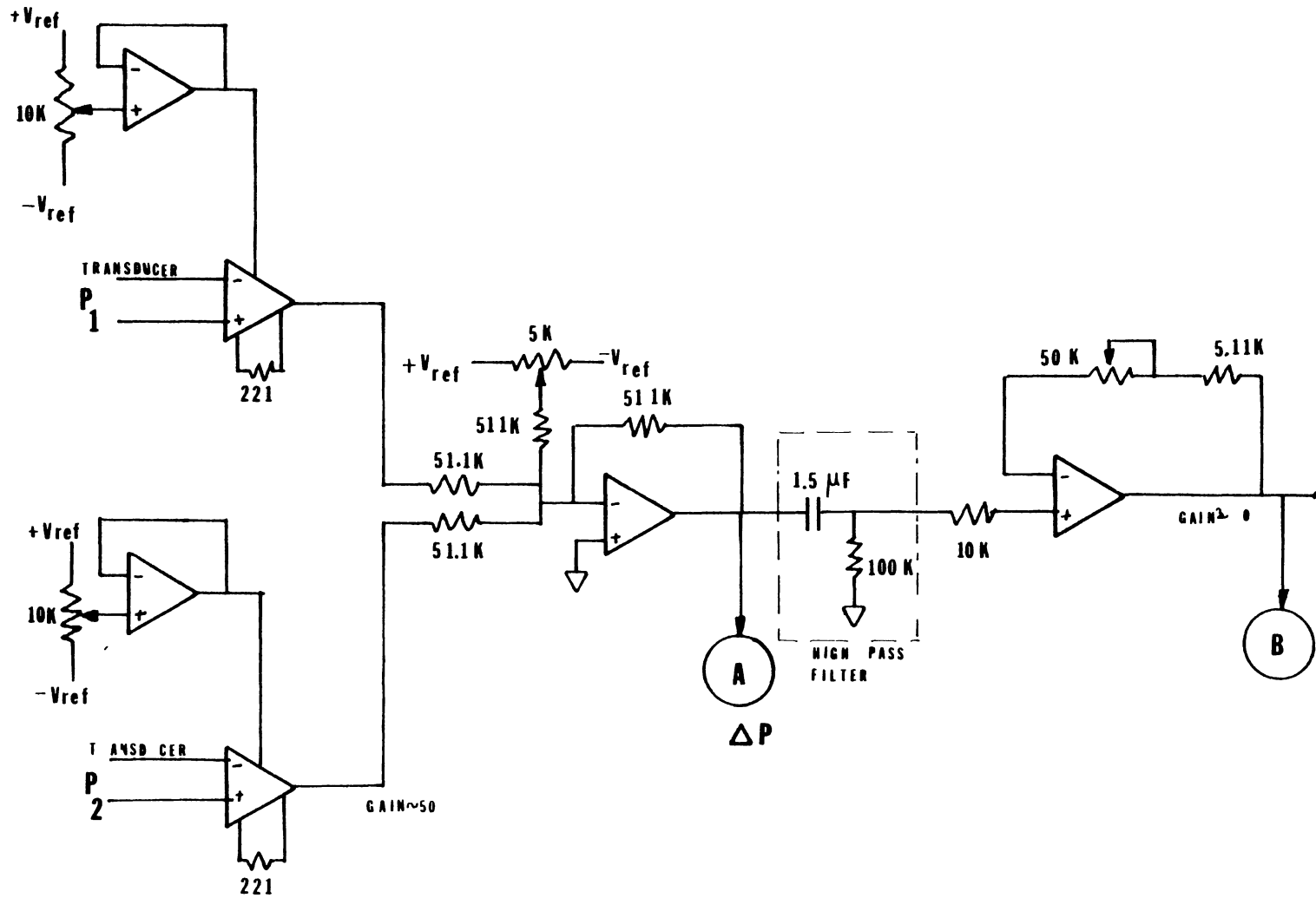


Figure 21. Pressure Measurement Schematics.

point A), and eliminates 60HZ and 400HZ noise corruption. This signal is transferred via a high pass filter (1.1 HZ break frequency) and amplified by a gain of about 3.0 (Test point B).

The measurements from test point A and test point B are given in Figure 22 for three identical step inputs to the system. (More identical runs were done with the same results). The recorder chart speed was 100 mm/sec and the full scale of the output was 5 volts.

Figure 23 is an enlargement (x4) of test point A recordings for the three runs, and Figure 24 is an enlargement (x3) of test point B recordings. By comparing the three runs, it can be observed that no appreciable random error or nonrepeatability exists in these responses. All three graphs are identical except for a small shift in run no. 2. This shift can be explained by the deviation of the starting point. The three axis motion platform by itself is a nonlinear system. Thus a small change of the starting point results in a different geometry and force balance. There is also a small nonlinearity in the chart recorder.

#### Measurement error prediction

The following data were taken from the instrument specifications:

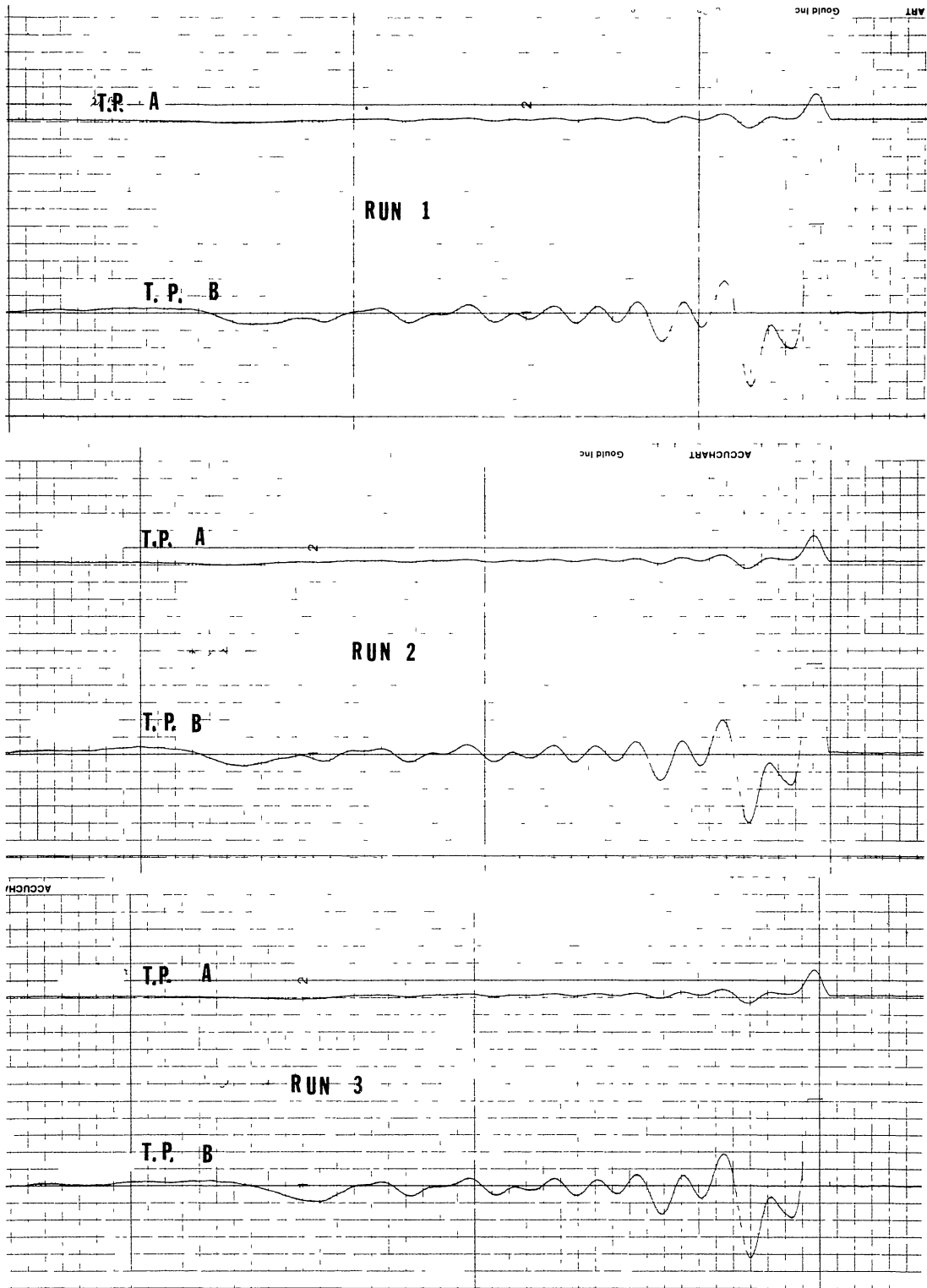


Figure 22. Pressure Measurement Test Results.

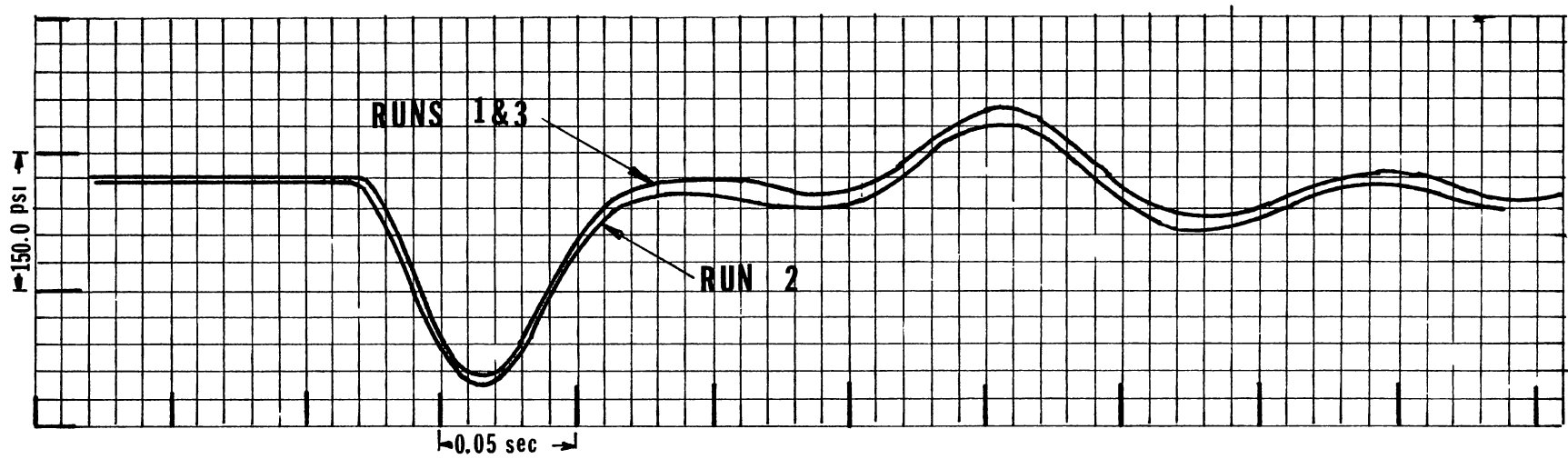


Figure 23. Test point A Recording for three runs  
(Enlargement X4)



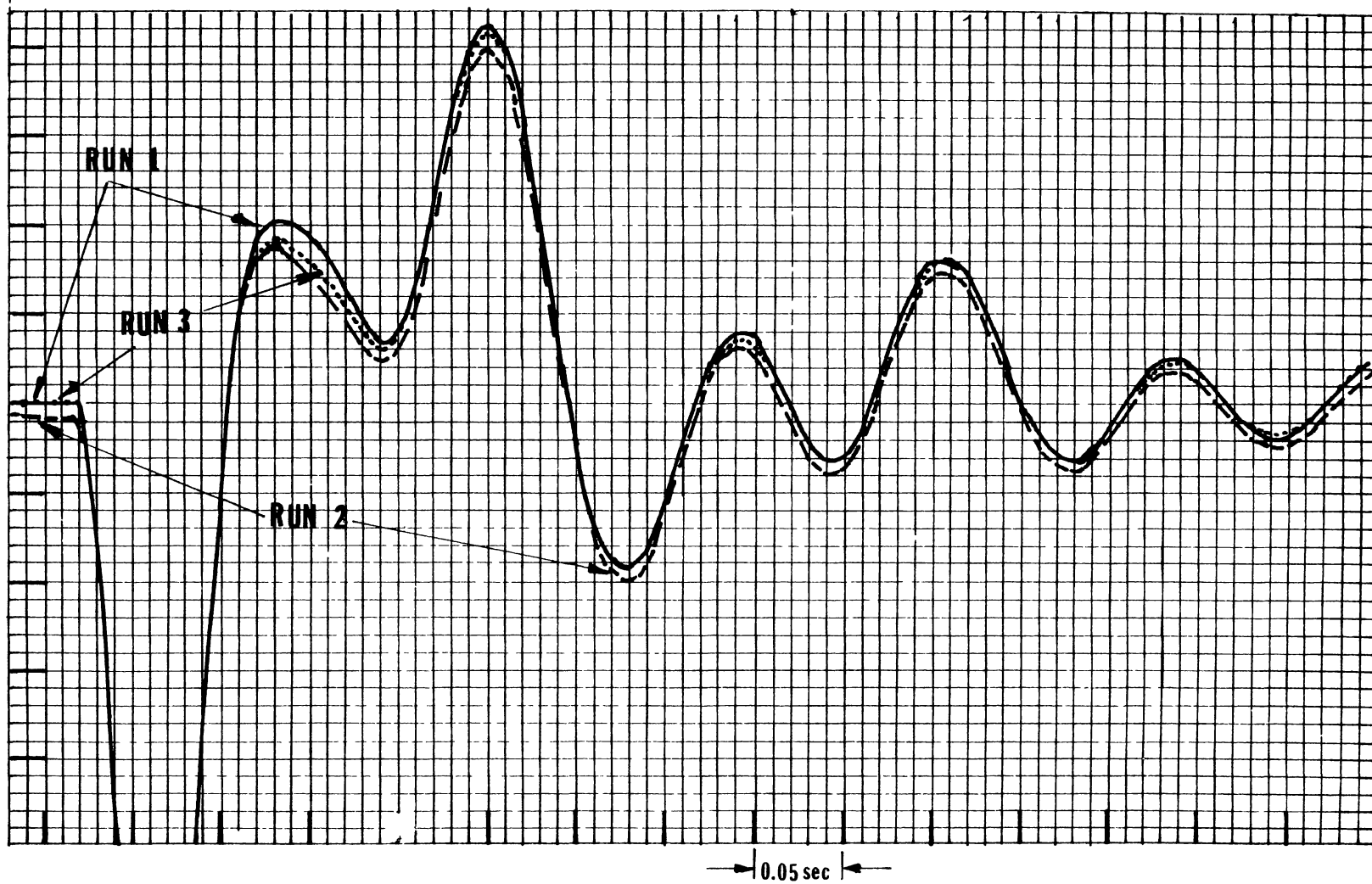


Figure 24. Test point B Recording for three runs  
(Enlargement X3)

Pressure Transducer: (MOOG Model 131-117):

Natural Frequency: 40 KHZ  
 Rise Time (90% F.S.): 0.4 msec  
 Hysteresis:  $\pm .25\%$  full scale  
 Repeatability:  $< 0.1\%$   
 Static error:  $< 0.25\%$

Displacement/Velocity Transducer

(Tempsonics Series DCTM)

Non-linearity:  $< 0.05\%$   
 Repeatability:  $< \pm 0.02\%$

Tempsonics claims that 60 HZ or 400 HZ noise will be rejected by the narrow band-width detector utilized in the transducer.

Gould Chart Recorder (Model 2400S)

Rise time to 40mm:  $< 4$  msec with 1% overshoot  
 Noise: 0.1 mm peak to peak  
 Attenuation: 0.99 (30 HZ)  
 0.98 (50 HZ)  
 0.9 (125 HZ)  
 0.707 (140HZ)  
 Chart speed:  $100 \pm 0.25$  mm/sec.  
 50 division per channel (1 mm/division)  
 Non-linearity:  $\pm 0.35\%$  full scale  
 Chart wander:  $\pm 0.25$  mm

Actuator friction:            7.6 Lbf

(the equivalent value in differential pressure is

$$\frac{7.6}{8.295} = 0.916 \text{ psi})$$

Accumulating the Data above

(in terms of pressure variables):

Frequency response for chart recorder ~1.0 psid

Chart wander = ±0.25 mm = ±7.5 psid

Recorder nonlinearity(50mm ~1500psi) =

$$1500 \times 0.35\% = 5.25 \text{ psid}$$

Pressure transducer repeatability = 0.1% x 3000.0=3psid

Friction in actuator ~1 psid

The overall error expected for the chart recording:

$$e = \sqrt{5.25^2 + 7.5^2 + 3^2 + 1^2 + 1^2} = 9.73 \text{ psid}$$

The overall error expected for the pressure measurement alone:

$$e = \sqrt{3^2 + 1^2} = 3.1 \text{ psid}$$

The overall error expected in measuring the differential pressure is 3.1 psid, plus the static error which may be as large as 0.25% (7.5 psi). But since the identification process needs only incremental values from the previous measurement, such an error will not constitute a problem.

Furthermore, during large pressure transients, the pressure may be changed by up to 60 psi per 0.02 sec. integration interval (see Figure 23).

## CHAPTER V

### SUMMARY AND RECOMMENDATIONS

The purpose of this research was to develop a new synthesis method for adaptive control of nonlinear dynamic systems with application to a one-degree-of-freedom motion simulator.

Digital computer capabilities along with existing control theories and reported optimization algorithms were integrated to develop a new approach to the solution of the control problem for a broad class of systems where actuator nonlinearities and dynamics dominate the system response.

It was demonstrated that for the class of systems studied, the controller design is substantially different if actuator dynamics are included rather than ignored. The adaptive control scheme leads to substantially improved performance compared to the conventional "fixed gain" design.

The method developed is especially useful for control problems where the frequencies of the inputs to the system are close to the natural frequencies of the system and a multi-input multi-output configuration is required.

#### 5.1 Summary

In section 1.1, the requirements of a motion platform

were given. The important problem of producing the washout motion needed to limit cues because of physical constraints was defined. Another problem identified was that the servo-actuator states were not employed in previous work (Ref. 2-11) in the design of control systems for motion platforms.

The potential for the utilization of digital computers in closed-loop electrohydraulic control systems was discussed in section 1.2. A conclusion was reached that a good application of computers in such systems may be in adaptive control. Methods of nonlinear control system synthesis were discussed in brief in section 1.3, and the recommendation of Kaufman and Teavassos of using direct gain optimization and direct estimation algorithm was given.

Section 1.5 evaluated the need for the development of an integrated, adaptive control scheme that includes the dominant nonlinearities and actuator subsystem dynamics in the plant model.

Methods and procedures used in the research to develop the adaptive control scheme were discussed in chapter II. The nonlinear identification and control algorithm developed in this chapter utilized the nonlinear process equations directly. The direct gain optimization and direct parameter optimization procedure used the Q-N (no line search) minimization algorithm reported by Davidon (25). The washout algorithm receives information (position, velocity), performs

decisions (when to switch mode of operation) and the input to the control system, based on the system response prediction is calculated.

Chapter III was devoted to the application model study. A general hardware configuration for a typical one degree-of-freedom motion platform was established and the nonlinear equations of motion were derived.

These equations were linearized about a steady-state operating point, and linearized control law was developed based upon the excess pole specification method (14). By comparing response characteristics of the linearized versus nonlinear model for the same control law, it was decided that controller gains derived from the linearized system of the case chosen are suitable to be initialized in the nonlinear adaptive controller.

The methods discussed in chapter II were employed to develop an adaptive control law for the application example. An off-line simulation program was written and executed on SEL 32/27 minicomputer (at Burtek Inc., Tulsa, OK). Three runs were made to test the response characteristics of the control system to three different time-dependent inputs. The first was low frequency acceleration input; the second was high frequency acceleration input, and the third was ramp input used to test the washout algorithm. The results of the simulation were evaluated against a traditional closed-loop electrohydraulic servo design. It was shown

that the proposed method improves system performance in terms of attenuation and phase lag for a wide spectrum of inputs, subject to hardware constraints. It eliminates the need for the trial-and-error process often used in tuning control system gains during hardware integration and eliminates the need for periodic adjustment of these gains due to drifts. The inclusion of the actuator system dynamics in the plant model and the development of an integrated control law simplify the control synthesis: The acceleration (force) control law was calculated directly from pressure measurements.

The control law devised in this thesis is adaptive in the sense that the control mode can be changed from a follower mode to a washout mode and vice versa automatically. The use of a separate washout mode eliminates the need for washout filters.

Implementation feasibility is discussed in chapter IV. Computer memory requirements for the application chosen look modest, about 5K-bytes of memory for on-line execution. Sequential processing for a one degree-of-freedom motion platform is possible. Execution estimates for the application model under study using an add/subtract time of 0.3 microsec., 10 integration steps; was 0.035 sec (AM4 integration method) or 0.016 sec. (AB2 integration method).

In order to predict whether noise might constitute a problem for the application under study, a test was conduc-



ted using existing motion platform hardware (of a C141B simulator), and no noticeable independent random noise was recorded.

Simulations of the proposed algorithm were executed using a data base word size of 32 bits. Since commercial A/D and D/A converters have up to 16 bits resolution (0.0015% accuracy), the data base word size for an on-line execution should be chosen accordingly. It is also recommended that the step size for on-line integration should be based upon the truncation error that will be determined by off-line simulation, employing a variable step size/variable order integration routine. The truncation error permitted by this selection should not be smaller than the accumulated dynamic measurements error (problem dependent).

Another recommendation, based on the identification results in Section 3.4.3 is that a "dither" signal input be used to excite the system, not only to minimize valve "stiction" effects, but to improve the identification process for parameters that affect the spring constant of the actuator (e.g., bulk modulus of the fluid).

## 5.2 Areas for Further Research

Areas recommended for future research are as follows:

1. Work is needed to define the acceptable limit of noise (and related number of integration steps) for which the method of this thesis can be used.

2. A quantitative exit criterion is needed for the optimization routine that will both suppress instability and conserve execution time.

3. The application example studied in this thesis was a one degree-of-freedom motion platform (one control, four states). Motion platforms with multiple degree-of-freedom have a multi input/multi output configuration. While the modelling of such a system with actuator dynamics incorporated is no major problem, research is needed into the parallel processing of the identification and control phases for such a case, so that all computations can be performed during run time.

It may be that the solution to such problems is the use of a high-speed array processor. For example, the specifications of the NUMERIX MARS 432 array processor include an add or multiply time of 0.1 microsecond, a real/real vector multiply time of 0.2 microsecond and a square root calculation time of 0.8 microseconds.

4. This research was conducted using the lumped parameter approach: This approach is justified when the servo-valve is mounted on the actuator, and the connecting line dynamics are negligible. An interesting area for research is the control of a distributed parameter system. K.N.Reid (28) discussed dynamic models of fluid transmission lines, but work is needed in their application to control system design.

5. A feasibility study of implementation, cost and reliability of the proposed design should be made prior to hardware design.

It is hoped that this research will motivate future research and design in the area covered by this thesis, and will enhance the solution of control problems involving systems which employ hydraulic actuation.

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APPENDIXES

## APPENDIX A

### ALGORITHM FOR UNCONSTRAINED MINIMIZATION

From the material discussed in Chapter II it is evident that an efficient algorithm for minimization of a nonlinear function, called the performance cost function must be employed. In the literature, the subject is discussed as nonlinear optimization or nonlinear programming.

The optimization problem will be of the form

minimize  $f(x)$ , subject to  $x \in \Omega$  where  $f$ ,

is a real-valued function and  $\Omega$ , the feasible set, is a subset of  $E^n$ . For the completely unconstrained case,  $\Omega = E^n$ .

Usually optimization problems have constraints, like the differential constraints ( $\dot{x} = f(x,t)$ ) of the dynamic system; however, some of the most powerful and convenient methods of solving constrained problems involve the conversion of the problem to one of unconstrained minimization.

#### A.1 Some Properties of a Minimum

The usual notion of a minimum is a point where a function has its least value, i.e.  $x_m$  such that  $F(x_m) \leq F(x)$



for all  $x$ .

For a function of an  $n$  variable  $x \equiv (x_1, x_2, \dots, x_n)$  with continuous derivatives, the minimum will be a point where

$$\frac{\partial F}{\partial x_i} = 0 \quad i = 1, 2, \dots, n \quad (\text{A.1-1})$$

A point satisfying the last equation is guaranteed to be a relative minimum if the Hessian matrix of  $F$  is positive definite (all eigenvalues are positive). The Hessian matrix is defined:

$$J = \begin{bmatrix} \frac{\partial^2 F}{\partial x_1^2} & \frac{\partial^2 F}{\partial x_1 \partial x_2} & \text{-----} & \frac{\partial^2 F}{\partial x_1 \partial x_n} \\ \vdots & \vdots & & \vdots \\ \frac{\partial^2 F}{\partial x_n \partial x_1} & \frac{\partial^2 F}{\partial x_n \partial x_2} & \text{-----} & \frac{\partial^2 F}{\partial x_n^2} \end{bmatrix} \quad (\text{A.1-2})$$

Geometrically, this property indicates that the quadratic function that approximates the original function at the point has its minimum there. Consider the Taylor series expansion of  $F$  about  $X_m$  up to quadratic terms:

$$F(x) = F(X_m) + \sum_{i=1}^N \left( \frac{\partial F}{\partial x_i} \right)_{X_m} (x_i - x_{mi}) + \frac{1}{2} \sum_{j=1}^N \sum_{i=1}^N \left( \frac{\partial^2 F}{\partial x_i \partial x_j} \right) (x_i - x_{mi}) (x_j - x_{mj}) \quad (\text{A.1-3})$$

In matrix form this becomes:

$$\begin{aligned}
 F(x) \simeq & F(X_m) + (X - X_m)^T \nabla F(X_m) \\
 & + \frac{1}{2}(X - X_m)^T J_m (X - X_m)
 \end{aligned}
 \tag{A.1-4}$$

The problem of relative minima is one of the most vexing in optimum methodology. This is because most of the viable methods can seek only a relative minimum and cannot converge in the general case to the least minimum.

## A.2 Gradient Methods for Minimization

### A.2.1 The Gradient

Central to these methods is the concept of the gradient of the function being minimized. This vector, denoted  $\nabla F$ , lies in the direction of greatest rate of change of the function and has that rate of change as its magnitude. The gradient of the function  $F(X)$  is defined as

$$\nabla F \equiv G \equiv \left( \frac{\partial F}{\partial X_1}, \frac{\partial F}{\partial X_2}, \dots, \frac{\partial F}{\partial X_n} \right)
 \tag{A.2.1-1}$$

### A.2.2 Line Search

Consider any vector  $S_q$  and the move prescription

$$X = X_q + \alpha S_q
 \tag{A.2.2-1}$$

Then if  $\alpha$  is considered a variable, the locus of  $X$  for a range of values of  $\alpha$  is a straight line. Substitute this formally in  $F(X)$ .

$$F(X) + F(X_q + \alpha S_q) = F(\alpha) \quad (\text{A.2.2-2})$$

Since  $F$  can be considered a function of  $\alpha$  alone ( $X_q$  and  $S_q$  are fixed). The value of  $\alpha$  which minimizes  $F(\alpha)$  is denoted  $\alpha^*$  and can be obtained by differentiating

$$\frac{dF}{d\alpha} = 0 \quad (\text{A.2.2-3})$$

subject to

$$\frac{d^2 F(\alpha)}{d\alpha^2} \Big|_{\alpha=\alpha^*} > 0 \quad (\text{A.2.2-4})$$

This process is called a line search and usually is done by quadratic or cubic interpolation.

### A.2.3 Gradient Algorithms

The procedure for function minimization by gradient methods is described in the flowchart of Figure 25. A few of the methods are given below.

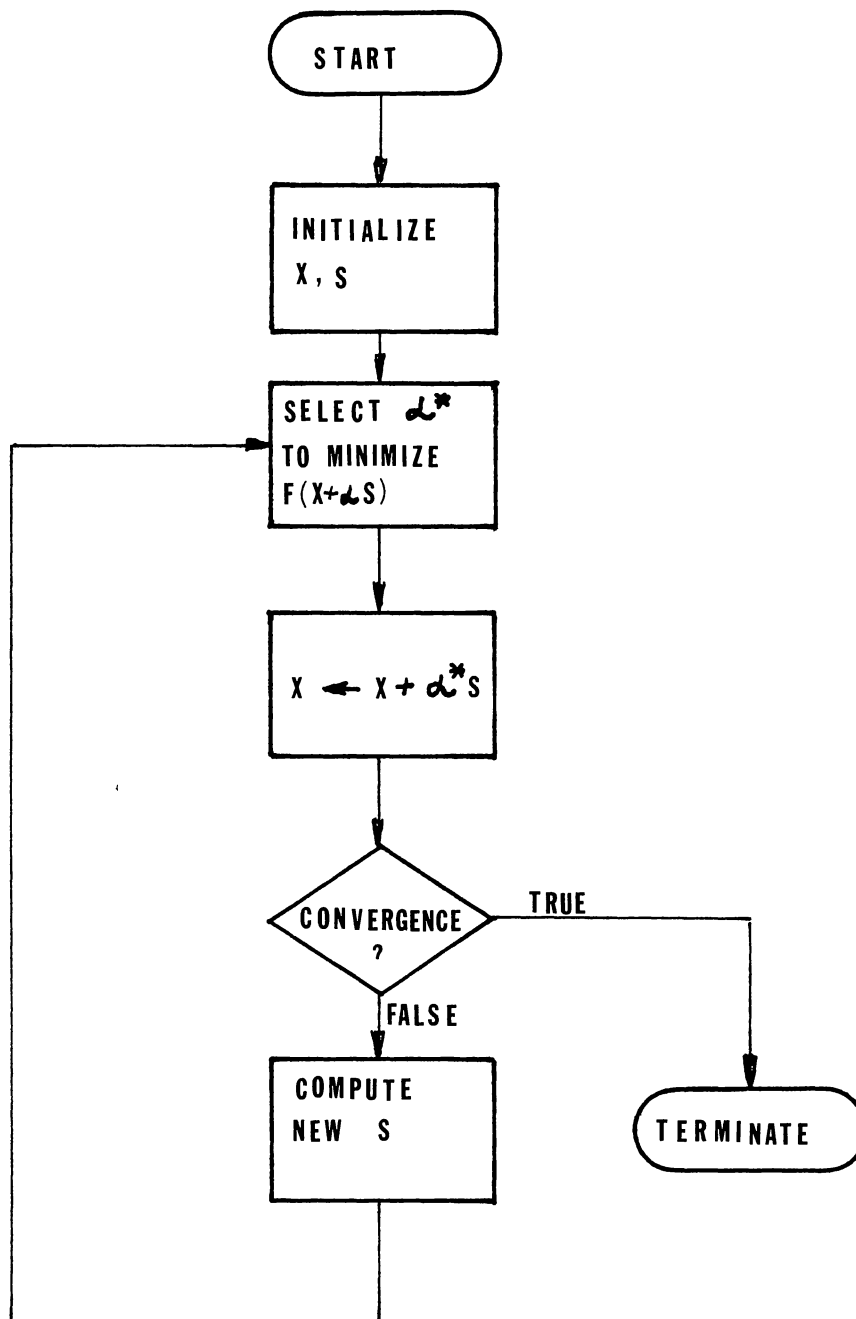


Figure 25. Gradient Algorithm Flow Chart.

A.2.3.1 Newton's Procedures. Expanding the value of  $F$  by Taylor series expansion up to the quadratic term

$$F(x) = F(X_q) + \nabla F_q^T (X - X_q) + \frac{1}{2} (X - X_q)^T J_q (X - X_q) \quad (\text{A.2.3.1-1})$$

Where  $J_q$  is the matrix of second partial derivatives of  $F$  evaluated at  $X_q$ . If  $\bar{F}_q(X)$  is an approximation to the minimum of  $F(X)$ , then  $X$  should satisfy the vector equation

$$\nabla \bar{F}_q = \nabla F_q + J_q (X - X_q) = 0$$

or

$$J_q X = J_q X_q - \nabla F_q \quad (\text{A.2.3.1-2})$$

multiplying through by the inverse of  $J_q$ ,

$$X_{q+1} = X_q - J_q^{-1} \nabla F_q \quad (\text{A.2.3.1-3})$$

Since  $F$  usually is not quadratic in  $X$ , the process can be improved by selecting

$$S = J_q^{-1} \nabla F_q \quad (\text{A.2.3.1-4})$$

and producing a line search to get

$$X_{q+1} = X_q - \alpha_q^* s \quad (\text{A.2.3.1-5})$$

A.2.3.2 Davidon-Fletcher-Powell Method. (a quasi-Newton method). In this method the local Hessian  $J_q^{-1}$  is replaced by an approximate metric  $H_q$ . This substitution eliminates the need for evaluating second derivatives and performing matrix inversions.

Algorithm:

1. Start with  $X_0$  and  $H_0 = I$

$$\text{set } S_0 = -H_0 \nabla F_0$$

2. Compute  $X_{q+1} = X_q + \alpha_q^* S_q$

where  $\alpha_q^*$  minimizes  $F(X_q + \alpha S_q)$ .

3. Compute  $H_{q+1} = H_q + M_q + N_q$

where

$$Y_q = G_{q+1} - G_q = \nabla F(X_{q+1}) - \nabla F(X_q)$$

$$M_q = \alpha_q^* \frac{S_q S_q^T}{S_q^T Y_q}$$

$$N_q = - \frac{(H_q Y_q) (H_q Y_q)^T}{Y_q^T H_q Y_q}$$

4. Compute  $S_{q+1} = -H_{q+1} G_{q+1}$

and repeat from step 2.

A.2.3.3 Davidon's optimization algorithm (without line

search). A representation of the algorithm was given by Davidon (Ref 25). M. J. D. Powell<sup>1</sup> has written: "Numerical experiments with Davidon's algorithm indicate that it may be the best numerical method for calculating the least value of a differentiable function of several variables".

### Algorithm

Start evaluate  $f(x)$ ,  $g(x)$

set:  $X_0 = X$   $f_0 = f$   $k_0 = J^T g$   $w = k_0$

1. Define  $s = -k_0$ ,  $f_0 = k_0^T s$ ,  $\lambda = 2$

If  $4f_0 \geq -f'_0$  go to step 2.

If not, set  $s = -S_4 f_0 / f'_0$

and  $f'_0 = -4f_0$

2. Set  $X = X_0 + JS$

If  $-f'_0 < \epsilon$  stop. If not evaluate  $f(X)$ .

If  $f < f_0$  go to step 3

If not set  $S = S/2$ ,  $f'_0 = f'_0/2$ ,  $\lambda = 0.5$

and repeat this step.

3. Evaluate  $g(x)$ . Set  $k = J^T g$ ,  $f' = k^T s$

$b_0 = f' - f'_0$ ,  $m = s + k_0 - k$ ,  $X_0 = X$ ,  $f_0 = f$ ,  $k_0 = k$ ,  $f'_0 = f'$

If  $b_0 \geq \epsilon$  go to step 4

If not, set  $S = S\lambda$ ,  $f'_0 = f'_0 \lambda$  and return to

step 2.

---

<sup>1</sup>Powell, M. J. D. 1977. Quadratic Termination Properties of Davidon's New Variable Metric Algorithm. Math Programming 12, pp. 141-147.

4. Define  $m^2 = m^T m$   
 If  $m^2 < \epsilon$  return to step 1  
 Else, define:  $v = m^T s, \mu = v - m^2, u = w - m m^T w / m^2$   
 If  $10^6 (m^T u)^2 < m^2 u^T u$  go to step 4a, else,  
 set  $n = m \times 1$  null vector,  $n^2 = 0$  and go to step 5
- 4a. set  $n = u u^T s / u^T u, n^2 = (u^T s)^2 / u^T u$
5. Set  $b = n^2 + \mu v / m^2$   
 if  $b \geq \epsilon$  go to step 6  
 Else  $n = s - m v / m^2, n^2 = b_0 - \mu v / m^2, b = b_0$
6. If  $\mu v < m^2 n^2$  go to step 6a  
 Else set:  $\gamma = 0, \Delta = (v / \mu)^{\frac{1}{2}}$  and go to step 7
- 6a. Set  $a = b - \mu, c = b + v, \gamma = \sqrt{\frac{1 - \mu v / m^2 n^2}{ab}}$   
 and  $\Delta = \sqrt{c/a}$   
 If  $c \geq a$  go to step 7, else, set  $\gamma = -\gamma$
7. Set:  $\alpha = v + \mu \Delta + m^2 n^2 \gamma$   
 $P = m(\Delta - n^2 \gamma) + n \gamma v$   
 $q = m(1 + n^2 \gamma) / \alpha - n \gamma \mu / \alpha$   
 $w = m n^2 (1 + \gamma \mu v / \alpha) - n(1 + \Delta) \mu v / \alpha$   
 $k_0 = k_0 + p q^T k_0, J = J + J_q p^T$   
 If  $n^2 > 0$  return to step 1. If not, set  $w = k_0$   
 and return to step 1



## APPENDIX B

### NUMERICAL INTEGRATION METHODS

#### USED IN SECTION 5.2

##### B.1 Runge-Kutta Methods

Integration methods may be classified as either single-step or multi-step methods. One important group of single-step algorithms are the Runge-Kutta (RK) methods. For differential equations of the form  $dx/dt = f(x,t)$ , RK methods require the evaluation of  $f(x,t)$  at two, three and four values of  $t$  on the interval  $t_i \leq t \leq t_{i+1}$  for second, third and fourth order approximations, respectively.

##### B.1.1 Second-order Runge-Kutta method

$$X_{i+1} = X_i + \frac{h}{2}(m_0 + m_1) \quad (\text{B.1.1-1})$$

where  $h$  is the step size and

$$\begin{aligned} m_0 &= f(x_i, t_i) \\ m_1 &= f(x_i + m_0 h, t_{i+1}) \end{aligned} \quad (\text{B.1.1-2})$$

The expression for  $m_1$  in equation (B.1.1-2) uses a predicted

value of  $x$  at the end point of the interval. The corrector equation (B.1.1-1) utilizes the average between the initial slope ( $m_0$ ) and a predicted slope ( $m_1$ ) for proceeding over the entire interval.

#### B.1.2 Fourth-order Runge-Kutta method (RK4)

$$X_{i+1} = X_i + \frac{h}{6}(m_0 + 2m_1 + 2m_2 + m_3) \quad (\text{B.1.2-1})$$

where,

$$\begin{aligned} m_0 &= f(x_i, t_i) \\ m_1 &= f\left(x_i + \frac{m_0 h}{2}, t_i + \frac{h}{2}\right) \\ m_2 &= f\left(x_i + \frac{m_1 h}{2}, t_i + \frac{h}{2}\right) \\ m_3 &= f(x_i + m_2 h, t_i + h) \end{aligned} \quad (\text{B.1.2-2})$$

### B.2 Multi-step Methods

Integration formulas which require information not only at  $t_i$  but also outside the integration interval under consideration ( $t_i, t_{i+1}$ ), are referred to as multi-step methods. A disadvantage of these methods is the requirement for additional information to start the procedure. However these methods usually require considerably less computational time than single-step methods.

### B.2.1 Adams-Bashforth second order

#### Predictor (AB2)

This multi-step method utilizes a single past slope:

$$X_{i+1} = X_i + \frac{h}{2}(3\dot{x}_i - \dot{x}_{i-1}) + \frac{5}{12} h^3 \ddot{x}(\xi_i) \quad (\text{B.2.1-1})$$

### B.2.2 Adams-Moulton (Predictor-corrector)

#### fourth order (AM4)

$$\text{Predictor: } X_{i1} = X_i + \frac{h}{24}(55\dot{x}_i - 59\dot{x}_{i-1} + 37\dot{x}_{i-2} - 9\dot{x}_{i-3})$$

$$\begin{aligned} \text{Corrector: } X_{i+1} = X_i + \frac{h}{24}(9\dot{x}_{i1} + 19\dot{x}_i - 5\dot{x}_{i-1} + \dot{x}_{i-2}) \\ - \frac{19}{720}h^5 \ddot{x}(\xi_i) \end{aligned} \quad (\text{B.2.2-1})$$

## B.3 Selecting the Step Size for Integration

The step size for integration may be selected using a predicted truncation error. This error is usually defined as the difference between the predictor and corrector formulas. This selection is method- and problem-dependent, and needs to be evaluated for the specific application. Kaufman and Teavassos (20) proposed the following procedure for the predictor/corrector equations

$$Y_{i+1}^P = Y_{i-1}^C + \frac{h_i}{3}(8f_i^P - 5f_{i-1}^C + 4f_{i-2}^C - f_{i-3}^C) \quad (\text{B.3.1})$$

$$Y_i^C = Y_{i-1}^C + \frac{h_i}{24}(9f_i + 19f_{i-1}^C - 5f_{i-2}^C + f_{i-3}^C)$$

(B.3.2)

The local truncation error associated with the Adams-Moulton corrector is

$$d_{i,c} \triangleq Y(t_i) - Y_i^C = -\frac{19}{720}h^5 y^{(5)}(\xi_i), \xi_i \in [t_0, t_i]$$

(B.3-3)

By definition the local truncation error associated with the predictor is given by

$$d_{i+1,p} \triangleq Y(t_{i+1}) - Y_{i+1}^P$$

(B.3-4)

Assuming

$$Y(t) = t^5, t \geq 0.0 \quad \text{and} \quad Y' = f(y,t) = 5t^4$$

and substituting into the predictor equation (B.3-2)

$$d_{i+1,p} = \frac{232}{720} h^5 y^{(5)}(\xi_2)$$

(B.3-5)

Then, assuming  $\xi_1 = \xi_2 = \xi$

$$\begin{aligned}
 d_{i+1,p} - d_{1,c} &= Y(t_{i+1}) - Y^P_{i+1} + (Y^C_i - Y(t_i)) \\
 &= \frac{251}{720} h^5 Y^{(5)}(\xi)
 \end{aligned}
 \tag{B.3-6}$$

Using these results

$$d_{i,c} = -\frac{19}{251} [Y^C_i - Y^P_{i+1} + \delta_Y] \tag{B.3-7}$$

where

$$\delta_Y = Y(t_{i+1}) - Y(t_i)$$

Using the Schwartz and triangle inequality

$$|d_{i,c}| \leq \frac{19}{251} \left\{ |Y^C_i - Y^P_{i+1}| + |\delta_Y| \right\}$$

By approximating  $|\delta_Y| = h_i |f[y(t_i), t]| + |O(h_i^2)|$ ,

$$\text{then } d_{i,c} \leq \frac{19}{251} \left\{ |Y^C_i - Y^P_{i+1}| + h_i |f(Y^C_i, t_i)| \right\} \tag{B.3-8}$$

The procedure that was advised by Kaufman and Teavassos was to evaluate the predictor corrector equations, then to estimate the local truncation error.

$$d_{ic} = \frac{19}{251} \left\{ \left| \bar{Y}_i^C - \bar{Y}_{i+1}^P \right| + h_i \left| f_i^C \right| \right\} \quad (\text{B.3-9})$$

if  $d_i > \epsilon_{\max}$ ; replace  $h_i \leftarrow h_i/2$   
 if  $d_i < \epsilon_{\min}$ ; replace  $h_i \leftarrow h_i \times 2$   
 else leave  $h_i$  .

At present, the most popular predictor-corrector algorithms control the truncation error by varying both the step size and the order of the method; and all of these algorithms use the Adams family of formulas.

APPENDIX C

LISTING OF SIMULATION TEST PROGRAM

```

$JOB B NLCNT2 AMOS SLOF=L NLCNT2          0001 000          DD 1017 I=1 3          0057 000
$OPTION 2 3 4 5                          0002 000          READ(5 1000)COM      0058 000
#FORT77                                   0003 000          1017 WRITE(1 1001)COM 0059 000
C                                          0004 000          READ(5 1003)N,ITERI 0060 000
C                                          0005 000          READ(5,1002)((FM(I J),I=1,N) J=1 N) 0061 000
C                                          0006 000          READ(5,1002)(GM(I) I=1 N) 0062 000
C          PROGRAM NLCNT                0007 000          READ(5,1002)BEGX,ENDX DELX ENDTIME 0063 000
C                                          0008 000          READ(5 1002)(RREF(I) I=1 N) 0064 000
C          THIS PROGRAM DEMONSTRATES NONLINEAR CONTROL SYSTEM SYNTHESIS 0010 000          READ(5,1002)(RK(I) I=1 N) 0065 000
C          WITH APPLICATION TO ELECTRO HYDRAULIC SERVO SYSTEM          0011 000          READ(5 1002)(Y(I),I=1 N) 0066 000
C                                          0012 000          READ(5 1002)(RGF1(I),I=1,N) 0067 000
C                                          0013 000          READ(5,1002)(RGF2(I) I=1 N) 0068 000
C          SUBROUTINES CALLED  RK41,GNDAV1 LXYPLT 0014 000          LPRINT= FALSE      0069 000
C                                          0015 000          LBUG= FALSE '*****FOR DEBUG ONLY ***** 0070 000
C                                          0016 000          C          0071 000
C          DEFINE LOCAL VARIABLES          0017 000          WRITE(1 200)        0072 000
C                                          0018 000          200 FORMAT( 0 10X, PLOT OF INPUT ACC OUT ACC POSITION CONTROL /) 0073 000
C          REAL*4      REPSILON ' MINIMIZATION LIMIT          0019 000          WRITE(1 210)BEGX ENDX DELX ENDTIME 0074 000
C          REAL*4      RRK1(4) ' TEMPORARY VALUE FOR GAINS          0020 000          210 FORMAT(5X START TIME= ,F7 3,5X TIME INTERVAL= F7 3 5X, 0075 000
C          REAL*4      RKFM(4) ' TEMPORARY VALUE FOR PARAMETERS          0021 000          X INTEGRATION STEP= F7 3 5X 'ENDTIME' F7 3//) 0076 000
C          REAL*4      ENDTIME ' END TIME OF SIMULATION          0022 000          C          0077 000
C          INTEGER*2    NS ' NUMBER OF VARIABLES FOR MINIMIZATION          0023 000          CALL LXYPLT(4 20,200 , -200 200 200 15 0 -15 0 40 -40 0078 000
C          INTEGER*2    ITERI ' NO OF ITERATION FOR IDENTIFIC          0024 000          X RREF(5),Y(5) Y(1) RCONT 0 05) 0079 000
C          INTEGER*2    ITERC ' NO OF ITERATIONS FOR CONTROL          0025 000          C          0080 000
C          INTEGER*2    NITER ' MAX ITERATION FOR MINIMIZATION          0026 000          CALL GNDAV? ' INITIALIZE MACHINE EPSILON 0081 000
C          LOGICAL*1    LPRINT ' PRINT FLAG(MINIMIZATION)          0027 000          C          0082 000
C          LOGICAL*1    LSYS ' FOR SYSTEM IDENTIF/CONTROL          0028 000          Y(5)=0 0 ' INITIALIZE REFERENCE INPUT 0083 000
C          LOGICAL*1    LBUG ' FOR DEBUG ONLY          0029 000          C          0084 000
C          LOGICAL *1    COM(65)          0030 000          C          0085 000
C                                          0031 000          C          0086 000
C                                          0032 000          C          ***** 0087 000
C          DEFINE COMMONS          0033 000          C          0088 000
C          COMMON/GM/REPSILON NITER LPRINT LSYS          0034 000          10 CONTINUE ' START LOOP HERE 0089 000
C          COMMON/GM/REPSILON NITER LPRINT LSYS          0035 000          C          0090 000
C          INCLUDE C CONTR?          0036 000          C          STORE STATES AT START OF INTEGRATION INTERVAL 0091 000
C          INCLUDE C DERF?          0037 000          C          0092 000
C          0038 000          C          DO 20 I=1,N+1 0093 000
C          0039 000          C          STRY(I)=Y(I) 0094 000
C          0040 000          20 CONTINUE 0095 000
C          0041 000          C          0096 000
C          0042 000          C          0097 000
C          0043 000          C          INTEGRATE THE NON-LINEAR EQUATIONS (PLANT) FROM BEGX TO ENDX 0098 000
C          0044 000          C          AND PLOT RESULTS 0099 000
C          0045 000          C          LINEAR=TRUE FOR UPDATE MODEL, =FALSE FOR NONLINEAR PLANT 0100 000
C          0046 000          C          0101 000
C          0047 000          C          LINEAR= FALSE ' REAL PLANT 0102 000
C          0048 000          C          LPLOTT= TRUE ' PLOT RESULTS 0103 000
C          0049 000          C          0104 000
C          0050 000          C          CALL RK41 ' INTEGRATION ROUTINE 0105 000
C          0051 000          C          0106 000
C          0052 000          C          LPLOTT= FALSE ' SUPPRESS PLOT FOR CONTROL GENERATION 0107 000
C          0053 000          C          0108 000
C          0054 000          C          ***** 0109 000
C          0055 000          C          0110 000
C          0056 000          C          0111 000
C          WRITE(1 1005)          0056 000          C          0112 000

```



```

04/11/84 16 57 22 TASK # 160000B4 AMUS GOULD S F L MPX-C 04/11/84 16 57 22 TASK # 160000B4 AMNS GOULD S E L MPX-3

C 0113 000 C 0169 000
C 0114 000 C 0170 000
C STORE LAST VALUES OF STATES AND REFERENCES 0115 000 DO 42 J=1,NS 0171 000
C 0116 000 RFM(J)=RKFM(J)+ 25*(RFM(J)-RKFM(J)) 0172 000
C 0117 000 42 CONTINUE 0174 000
RBEQX=BEGX 0118 000 C 0175 000
DELTA=ENDX-RBEQX 0119 000 C 0176 000
C 0120 000 C 0177 000
DO 30 I=1,N+1 0121 000 C RETURNED VALUE WILL BE UPDATED MODEL 0178 000
ENDY(I)=Y(I) 0122 000 C 0179 000
RREFX(I)=RREF(I) 0123 000 C 0180 000
30 CONTINUE 0124 000 C***** 0181 000
C 0125 000 C 0182 000
C STORE STATES, REFERENCES LAST 0 2*DELTA FOR IDENTIFICATION 0126 000 C 0183 000
C 0127 000 C CONTROL CALCULATION 0184 000
C 0128 000 C 0185 000
JL=JA 0129 000 C 0186 000
DO 50 I=1,N +1 0130 000 C 0187 000
DO 50 JK=1,JL/5 0131 000 NITER=ITERC ' MAX NO OF ITERATIONS FOR CONTROL 0188 000
RYXACT(I,JK)=YX(I,JL-JL/5+JK-1) 0132 000 C 0189 000
RREFC(I,JK)=RREFI(I,JL-JL/5+JK) 0133 000 C 0190 000
50 CONTINUE 0134 000 C OPTIMIZE CONTROL BASED ON UPDATED MODEL 0191 000
C 0135 000 C (CALCULATE NEW GAINS) 0192 000
C 0136 000 C 0193 000
EVALUATE FIRST,SECOND ,THIRD ORDER DERIVATIVE 0137 000 C LSYS= FALSE ' CONTROL FLAG 0194 000
AT CONTROL POINT FOR REFERENCE INPUT EXTRAPOLATION 0138 000 C 0195 000
C 0139 000 C 0196 000
RTANG(1)=(RREFI(5,JL)-2 0*RREFI(5,JL-1)+3 0*RREFI(5,JL))/2 0 0140 000 DO 31 J=1,4 0197 000
RTANG(2)=RREFI(5,JL)-2 0*RREFI(5,JL-1)+RREFI(5,JL-2) 0141 000 RRKI(J)=RK(J) 0198 000
RTANG(3)=RREFI(5,JL)-3 0*RREFI(5,JL-1)+3 0*RREFI(5,JL-2) 0142 000 31 CONTINUE 0199 000
X -RREFI(5,JL-3) 0143 000 C 0200 000
C 0144 000 C 0201 000
C 0145 000 C 0202 000
***** 0146 000 CALL GNDAVI(RK,N) ' OPTIMIZATION SUBROUTINE 0203 000
C 0147 000 C 0204 000
C 0148 000 C 0205 000
C UPDATE SYSTEM PARAMETERS 0149 000 DO 32 J=1,N 0206 000
C 0150 000 RK(J)=RRKI(J)+ 15*(RK(J) -RRKI(J)) 0207 000
C RFM( )-VARIABLES THAT MINIMIZE ERROR FUNCTION 0151 000 32 CONTINUE 0208 000
C RFM(1)-BULK MOD-FLUID,RFM(2)-VALVE GAIN 0152 000 C 0209 000
C 0153 000 C RETURNED VALUES WILL BE OF UPDATED GAINS 0210 000
C 0154 000 C 0211 000
REPSILON=0 1E-10 ' ACCURACY LIMIT 0155 000 C 0212 000
NS=2 ' NO OF VARIABLES FOR MINIMIZATION ROUTINE 0156 000 C 0213 000
NITER=ITERI ' MAX NO OF ITERATIONS FOR IDENTIFICATION 0157 000 C 0214 000
LPRINT= FALSE 0158 000 C ***** 0215 000
LSYS= TRUE ' IDENTIFICATION FLAG 0159 000 C 0216 000
C 0160 000 C 0217 000
C LINEAR= TRUE ' ADAPTIVE MODEL 0161 000 C PREPARE FOR NEW CYCLE 0218 000
C 0162 000 C 0219 000
DO 41 J=1,NS 0163 000 BEGX=RBEQX+DELTA 0220 000
RKFM(J)=RFM(J) 0164 000 ENDX=BEGX+DELTA 0221 000
41 CONTINUE 0165 000 DO 40 I=1,N+1 0222 000
C 0166 000 Y(I)=ENDY(I) 0223 000
C 0167 000 RREF(I)=RREFX(I) 0224 000
C 0168 000 40 CONTINUE 0225 000
CALL GNDAVI(RFM,NS) ' OPTIMIZATION SUBROUTINE

```

04/11/84 16 57 22 TASK # 16000084 AMOS GOULD S E L MPX-2

```
C                                0226 000
      IF(X LT ENDTIME)GO TO 10      ' ELSE STOP SIMULATION      0227 000
C                                0228 000
      PRINT VALUES OF GAINS,PARAMETERS AND REFERENCES      0229 000
      AT TERMINATION POINT      0230 000
C                                0231 000
C                                0232 000
      WRITE(1,901)(RK(I),I=1,4)      0233 000
901  FORMAT('0',5X,'K= ',4(E12 5,5X))      0234 000
      WRITE(1,902)(RFM(I),I=1,2)      0235 000
902  FORMAT(5X,'RFM= ',2(E12 5,5X))      0236 000
      WRITE(1,903)(RREFX(I),I=1,5)      0237 000
903  FORMAT(5X,'RREFX= ',5(E12 5,5X))      0238 000
C                                0239 000
      STOP      0240 000
      END      0241 000
%A1 LIB=HYDROLIB,,U      0242 000
%A1 DIR=HYDRODIR,,U      0243 000
$CATALOG      0244 000
A1 5=D CONT2      0245 000
A2 1=SLO,1000      0246 000
A2 LP=SLO,20000      0247 000
BUILD R NLCN12      0248 000
$EQJ      0249 000
**      0250 000
```

```

04/11/84 16 58 34 TASK # 16000084 AMOS GOULD S E L MPX-3c
%JOB B RK42 AMOS SLOF=L RK42 0001 000 I1=2*N 0057 000
%OPTION 2 3 4 5 0002 000 I2=3*N 0058 000
%FORT77 0003 000 C 0059 000
C 0004 000 C 0060 000
C 0005 000 C***** 0061 000
C 0006 000 C 0062 000
C 0007 000 C START LOOP HERE 0063 000
C SUBROUTINE RK41 0008 000 C 0064 000
C 0009 000 C 0065 000
C 0010 000 10 CONTINUE 0066 000
C 0011 000 C 0067 000
C 0012 000 C IF( NOT LINEAR)THEN ' OPERATION IS ON PLANT MODEL 0068 000
C 0013 000 C 0069 000
C 0014 000 C CHECK PLANT MEASUREMENTS AND DETERMINE IF NEEDED TO 0070 000
C 0015 000 C SWITCH BETWEEN TWO MODES OF OPERATION 0071 000
C 0016 000 C 0072 000
C 0017 000 C 0073 000
C 0018 000 C 0074 000
C 0019 000 C 0075 000
C 0020 000 C IF(ABS(Y(1)) LT 0 2)THEN 0076 000
C 0021 000 C LBRAKE= FALSE 0077 000
C 0022 000 C LREV= FALSE 0078 000
C 0023 000 C ELSE IF((Y(2) GT 0 0) AND (Y(1) GT 0 0))THEN 0079 000
C 0024 000 C RY1LIM=ABS(RXLIM-Y(1)) 0080 000
C 0025 000 C RY2LIM=SQRT(2 0*3B 0*RY1LIM) 0081 000
C 0026 000 C IF(Y(2) GT RY2LIM)LBRAKE= TRUE 'VELOCITY EXCEEDS LIMIT 0082 000
C 0027 000 C ELSE IF((Y(2) LT 0 0) AND (Y(1) LT 0 0))THEN 0083 000
C 0028 000 C RY1LIM=ABS(RXLIM+Y(1)) 0084 000
C 0029 000 C RY2LIM=SQRT(2 0*3B 0*RY1LIM) 0085 000
C 0030 000 C IF(ABS(Y(2)) GT RY2LIM)LBRAKE= TRUE 'VELOC EXCEEDS LIM1 0086 000
C 0031 000 C 0087 000
C 0032 000 C 0088 000
C 0033 000 C END IF 0089 000
C 0034 000 C 0090 000
C 0035 000 C 0091 000
C 0036 000 C RXR(JA)=X ' STORE GREED TIME 0092 000
C 0037 000 C 0093 000
C 0038 000 C 0094 000
C 0039 000 C *****INPUT SIGNAL GENERATION***** 0095 000
C 0040 000 C 0096 000
C 0041 000 C INPUT SIGNAL HERE WILL BE ACCELERATION 0097 000
C 0042 000 C 0098 000
C 0043 000 C 0099 000
C 0044 000 C 0100 000
C 0045 000 C GENERATION OF INPUT SIGNAL IN BRAKE/WASHOUT MODE 0101 000
C 0046 000 C 0102 000
C 0047 000 C 0103 000
C 0048 000 C 0104 000
C 0049 000 C IF(LBRAKE)THEN 0105 000
C 0050 000 C RRR=-SIGN(3B 0,Y(1)) 0106 000
C 0051 000 C LREV=LREV OR (ABS(Y(1)) GE RXLIM) 0107 000
C 0052 000 C 0108 000
C 0053 000 C IF(LREV)THEN 0109 000
C 0054 000 C RY1LIM=ABS(Y(1)) 0110 000
C 0055 000 C RY2LIM=SQRT(2 0*3B 0*RY1LIM) 0111 000
C 0056 000 C 0112 000

PURPOSE OF THE SUBROUTINE IS TO GENERATE INPUT SIGNAL TO THE
SYSTEM ,CALCULATE CONTROL AND INTEGRATE THE NON-LINEAR
EQUATIONS OF MOTION
INPUT IS GENERATED FOR TWO MODES FOLLOWER MODE AND WASHOUT
MODE INTEGRATION IS DONE BY FORTH ORDER RUNGE-KUTTA METHDD
THIS SUBROUTINE IS CALLED BY MAIN PROGRAM(NLCONT) AND BY
THE INDEX FUNCTION SUBROUTINES(FUNCT1,FUNCT2)
SUBROUTINES CALLED BY RK41 ARE THE SYSTEM EQNS OF MOTION
SUBROUTINES-DEFUN1 FOR UPDATE MODEL, DEFUN2 FOR PLANT MODEL

DECLARE COMMONS
INCLUDE C CONTR2
DECLARE LOCALS
REAL*4 RJA ' REAL(JA-1)
REAL*4 RXLIM ' LIMIT FOR TRAVEL
REAL*4 RY1LIM ' ABS (RXLIM-Y(1))
REAL*4 RY2LIM ' VEL LIMIT FOR BRAKE
REAL*4 WK(16 ) ' WORK VECTOR
INTEGER*1 IFUN ' CONTROL FOR PLOT
INTEGER*2 IC ' CONTROL FOR PLOT SPACING
LOGICAL*1 LBRAKE ' BRAKE IN EFFECT FLAG
LOGICAL*1 LREV ' MOTION EXCEEDS LIMIT
LOGICAL*1 LSYS ' IDENTIFICATION FLAG

COMMON/ON/LSYS

INITIALIZATION

JA=1
IC=0
RXLIM=10 0
IFUN =0
X=BEGX

```

```

IF((Y(1)*Y(2) LT 0 0) AND (ABS(Y(2)) GT 0 5*RY2LIM))THEN 0113 000
  IF(RREF(5)*RRR GT 0 0)THEN 0114 000
    RREF(5)=RREF(5)+ 05*SIGN(ABS(Y(2))-0 5*RY2LIM,Y(1)) 0115 000
  ELSE 0116 000
    RREF(5)=RREF(5)- 02*(RRR+RREF(5)) 0117 000
  END IF 0118 000
C 0119 000
C 0120 000
C 0121 000
ELSE 0122 000
  RREF(5)=RREF(5)+ 01*(RRR-RREF(5)) 0123 000
END IF 0124 000
C 0125 000
ELSE 0126 000
  RY1LIM=RXLIM-ABS(Y(1)) 0127 000
  RY2LIM=SQRT(2 0*38 0*RY1LIM) 0128 000
C 0129 000
C 0130 000
IF(RRR*RREF(5) LT 0 0)THEN 0131 000
  RREF(5)=RREF(5)- 10*SIGN(ABS(Y(2))-RY2LIM,Y(1)) 0132 000
ELSE 0133 000
  RREF(5)=RREF(5)+ 01*(RRR-RREF(5)) 0134 000
END IF 0135 000
C 0136 000
ELSE 0137 000
  END IF 0138 000
C 0139 000
C 0140 000
C 0141 000
C 0142 000
GENERATE INPUT SIGNAL FOR THE FOLLOWER MODE TO 0143 000
SIMULATE INPUT FROM AIRCRAFT EQUATIONS OF MOTION 0144 000
C 0145 000
C 0146 000
C 0147 000
RREF(5)=120 0*SIN(10 0*X) ' HIGH FREQUENCY INPUT 0148 000
RREF(5)=22 0*SIN(3 0*X) ' LOW FREQUENCY INPUT 0149 000
RREF(5)=80 0*X ' RAMP TO CHECK WASHOUT 0150 000
C 0151 000
END IF 0152 000
C 0153 000
C 0154 000
ELSE IF(LSYS)THEN ' *****IDENTIFICATION***** 0155 000
C 0156 000
C 0157 000
RREF(5)=RREFC(5, JA) ' FROM STORAGE 0158 000
C 0159 000
C 0160 000
ELSE ' *****CONTROL REFERENCE***** 0161 000
C 0162 000
C 0163 000
EXTRAPOLATE SIGNAL INPUT FOR CONTROL CALCULATIONS 0164 000
C 0165 000
RJA=JA-1 0166 000
RREF(5)=RREFX(5)+RTANG(1)*RJA+ 5*(RJA**2)*RTANG(2) 0167 000
X + 166667*(RJA**3)*RTANG(3) 0168 000

```

```

C 0169 000
END IF ' SIGNAL INPUT ESTABLISHED 0170 000
C 0171 000
C 0172 000
NOW CONVERT ACCELERATION INPUT 0173 000
TO PRESSURE VALUES 0174 000
C 0175 000
C 0176 000
C 0177 000
RREF(3)=1000 0+ 5*RREF(5)/ 926 0178 000
RREF(4)=1000 0- 5*RREF(5)/ 926 0179 000
C 0180 000
C 0181 000
DO 11 I=1,N +1 ' RCORD REFERENCE ON GREED 0182 000
  RREFI(I,JA)=RREF(I) 0183 000
CONTINUE 0184 000
C 0185 000
C 0186 000
C ***** 0187 000
C 0188 000
C 0189 000
RCONT=0 0 ' CONTROL CALCULATIONS 0190 000
DO 15 I=1,N 0191 000
  RCONT=RCONT+RK(I)*(RREF(I)-Y(I)) 0192 000
CONTINUE 0193 000
C 0194 000
IF(ABS(RCONT) GE 40 0)RCONT=SIGN(40 0 RCONT) 0195 000
C 0196 000
C 0197 000
C ***** 0198 000
C 0199 000
INTEGRATION STARTS HERE 0200 000
C 0201 000
C 0202 000
C 0203 000
***** EVALUATE DERIVATIVE(V1) 0204 000
IF(LINEAR)THEN 0205 000
  CALL DERFUN1(RCONT,N,Y,DY,RFM) ' UPDATED MODEL 0206 000
ELSE 0207 000
  CALL DERFUN2(RCONT,N,Y,DY,RFM) ' PLANT MODEL 0208 000
END IF 0209 000
C 0210 000
DO 20 I=1,N 0211 000
  WK(I)=Y(I)+DELX*DY(I)/2 0E00 0212 000
  WK(I+N)=DY(I) ' STORE V1 0213 000
CONTINUE 0214 000
C 0215 000
C 0216 000
C 0217 000
X=X+DELX/2 0E00 0218 000
***** EVALUATE V2 0219 000
IF(LINEAR)THEN 0220 000
  CALL DERFUN1(RCONT,N,WK,DY,RFM) 0221 000
ELSE 0222 000
  CALL DERFUN2(RCONT,N,WK,DY,RFM) 0223 000
END IF 0224 000

```

04/11/84 16 58 34 TASK # 16000084

AMOS GOULD S E L MPX-3

04/11/84 16 58 34 TASK # 16000084

AMOS GOULD S E L MPX-3.

```

C          0225 000
C          0226 000
DO 30 I=1,N
WK(I)=Y(I)+DELX*DY(I)/2 0E00
WK(I+1)=DY(I)          ' STORE V2
CONTINUE
30
C          0227 000
C          0228 000
C          0229 000
C          0230 000
C          0231 000
C          0232 000
C          0233 000
***** COMPUTE V3
IF(LINEAR)THEN
CALL DERFUN1(RCONT,N,WK,DY,RFM)
ELSE
CALL DERFUN2(RCONT,N,WK,DY,RFM)
END IF
C          0234 000
C          0235 000
C          0236 000
C          0237 000
C          0238 000
C          0239 000
C          0240 000
DO 40 I=1,N
WK(I)=Y(I)+DELX*DY(I)
WK(I+1)=DY(I)          ' STORE V3
CONTINUE
40
C          0241 000
C          0242 000
C          0243 000
C          0244 000
C          0245 000
C          0246 000
C          0247 000
C          0248 000
C          0249 000
C          0250 000
C          0251 000
C          0252 000
C          0253 000
C          0254 000
C          0255 000
C          0256 000
C          0257 000
C          0258 000
C          0259 000
C          0260 000
C          0261 000
C          0262 000
C          0263 000
C          0264 000
C          0265 000
C          0266 000
C          0267 000
C          0268 000
C          0269 000
C          0270 000
C          0271 000
C          0272 000
C          0273 000
C          0274 000
C          0275 000
C          0276 000
C          0277 000
C          0278 000
C          0279 000
C          0280 000

```

```

IF(LPLOT)THEN
IF(IC EQ 0)THEN
CALL LXYPLT(IFUN,20,200,-200,200,-200,15 0,-15 0
X 40 0,-40 0,RREF(5),Y(5),Y(1),RCONT,0 05)
IC=4
ELSE
IC=IC-1
END IF
C          0281 000
C          0282 000
C          0283 000
C          0284 000
C          0285 000
C          0286 000
C          0287 000
C          0288 000
C          0289 000
C          0290 000
C          0291 000
C          0292 000
C          0293 000
C          0294 000
C          0295 000
C          0296 000
C          0297 000
C          0298 000
C          0299 000
C          0300 000
C          0301 000
C          0302 000
C          0303 000
C          0304 000
C          0305 000
C          0306 000
C          0307 000
C          0308 000
C          0309 000
C          0310 000

```

```

*JOB B GNDV1 AMOS SLOF=L GNDV1
*OPTION 2 3 4 5
*FORT77
C
C SUBROUTINE TO FIND MINIMUM OF NONLINEAR FUNCTION
C OF N VARIABLES EMPLOYING QUASI-NEWTON METHOD OF
C W C DAVIDON(MATH PROGRAMMING,9(1975)1-30)
C WITHOUT LINE SEARCH
C SUBROUTINE GNDV1 (RVX,N) ' CONTROL VERSION
C
REAL*4 RMJ(20,20)
REAL*4 RVX(20)
REAL*4 RVX0(20)
REAL*4 RF ' FUNCTION VALUE
REAL*4 RFO ' F(0)
REAL*4 RVG(20) ' GRADIENT VECTOR
REAL*4 RVK0(20)
REAL*4 RVN(20)
REAL*4 RVS(20) ' DIRECTION VECTOR
REAL*4 RFO1
REAL*4 RLAMDA
REAL*4 REPSILON
REAL*4 RVK(20)
REAL*4 RF1
REAL*4 RBO
REAL*4 RVM(20)
REAL*4 RFM
REAL*4 RFMU
REAL*4 RFNU
REAL*4 RVU(20)
REAL*4 RVN(20)
REAL*4 RFN
REAL*4 RFB
REAL*4 RFGAMA
REAL*4 RFDDELTA
REAL*4 RFA
REAL*4 RFC
REAL*4 RMJT(20,20)
REAL*4 RTMP
REAL*4 RFALFA
REAL*4 RVP(20)
REAL*4 RVG(20)
REAL*4 X
REAL*4 B
REAL*4 A
REAL*4 EPS ' MACHINE EPSILON
REAL*4 ESP ' SQRT(MACH EPS)
REAL*4 RMTU ' 10-6(MU)2
REAL*4 RMTU
REAL*4 UTS
REAL*4 UTU
REAL*4 GTKO
REAL*4 SMGK(20,20)
REAL*4 D
INTEGER*2 IX ' CURRENT ITERATION #
INTEGER*2 NITER ' MAX # OF ITERATIONS
0001 000
0002 000
0003 000
0004 000
0005 000
0006 000
0007 000
0008 000
0009 000
0010 000
0011 000
0012 000
0013 000
0014 000
0015 000
0016 000
0017 000
0018 000
0019 000
0020 000
0021 000
0022 000
0023 000
0024 000
0025 000
0026 000
0027 000
0028 000
0029 000
0030 000
0031 000
0032 000
0033 000
0034 000
0035 000
0036 000
0037 000
0038 000
0039 000
0040 000
0041 000
0042 000
0043 000
0044 000
0045 000
0046 000
0047 000
0048 000
0049 000
0050 000
0051 000
0052 000
0053 000
0054 000
0055 000
0056 000
INTEGER*2 N ' #OF VARIABLES
LOGICAL*1 LPRINT ' TO PRINT EVERY ITERATION
LOGICAL*1 LSYS
COMMON/GN/ REPSILON, NITER,LPRINT,LSYS
C*****
C*****
C
C INITIALIZATION
C
C FUNCTION VAL FOR START
IF(LSYS)THEN
CALL FUNCT1(RF) ' IDENTIFICATION
ELSE
CALL FUNCT2(RF) ' FFEDBACK
END IF
J=[I]
DO 50 I=1,N
DO 50 J=1,N
C
IF(I NE J)THEN
RMJ(I,J)=0 0
ELSE
RMJ(I,J)=1 0
END IF
CONTINUE
50
C
GRADIENT
CALL GRAD1(RVX,N,RVG,ESP,LSYS)
L
C
RFO=RF
DO 60 I=1,N
RVK0(I)=0 0
DO 60 J=1,N
RVK0(I)=RVK0(I)+RMJ(I,J)*RVG(J)
CONTINUE
60
C
DO 70 I=1,N
RVX0(I)=RVX(I)
RVN(I)=RVK0(I)
CONTINUE
70
C
PRINT STARTING VALUES
WRITE(1,1010)
C010 FORMAT(10X,'INITIAL X = ')
WRITE(1,1012)(RVX(I) I=1 N)
0057 000
0058 000
0059 000
0060 000
0061 000
0062 000
0063 000
0077 000
0078 000
0079 000
0080 000
0081 000
0082 000
0083 000
0084 000
0085 000
0086 000
0087 000
0088 000
0089 000
0090 000
0091 000
0092 000
0093 000
0094 000
0095 000
0096 000
0097 000
0098 000
0099 000
0100 000
0101 000
0102 000
0103 000
0104 000
0105 000
0106 000
0107 000
0108 000
0109 000
0110 000
0111 000
0112 000
0113 000
0114 000
0115 000
0116 000
0117 000
0118 000
0119 000
0120 000
0121 000
0122 000
0123 000
0124 000
0125 000

```

04/11/84	16 59 16	TASK # 16000084	AMOS	GOULD S E L MPX-3	04/11/84	16 59 16	TASK # 16000084	AMOS	GOULD S E L MPX-3
C012	FORMAT(4X,4D18 10)		0126 000		C	CALL FUNCT(RF RVX,N)			0182 000
C	WRITE(1,1015)RF		0127 000		C	IF(LSYS)THEN			0183 000
C015	FORMAT(6X, INITIAL F VALUE = ',D18 10)		0128 000		C	CALL FUNCT1(RF) ' IDENTIFICATION			0184 000
C	WRITE(1,1020)		0129 000		C	ELSE			0185 000
C020	FORMAT(5X,'J-MATRIX,INITIALIZED = )		0130 000		C	CALL FUNCT2(RF) ' FEEDBACK			0186 000
C	WRITE(1,1012)((RMJ(I J),J=1,N),I=1,N)		0131 000		C	END IF			0187 000
C			0132 000		C				0188 000
C	INITIALIZE # OF ITERATIONS		0133 000		C	PRINT NEW VALUES FOR ITERATION POINT			0189 000
C			0134 000		C				0190 000
C	IX=0		0135 000		C	IF(LPRINT)THEN			0191 000
C			0136 000		C	WRITE(1,1050)IX,RF			0192 000
C	*****		0137 000		C050	FORMAT(/,5X, ITERATION = ',I3 3X, F= ' D18 10)			0193 000
C			0138 000		C	WRITE(1 1060)			0194 000
C	STEP NO 1		0139 000		C060	FORMAT(/ 5X,'X-VALUES= )			0195 000
C			0140 000		C	WRITE(1 1012)((RVX(I),I=1,N)			0196 000
1	CONTINUE		0141 000		C				0197 000
C			0142 000		C	WRITE(1,1070)			0198 000
C			0143 000		C070	FORMAT(/,5X, J-MTRIX = )			0199 000
C	RFO1=0 0		0144 000		C	WRITE(1,1012)((RMJ(I J) J=1 N) I=1 N)			0200 000
C	DO 80 I=1,N		0145 000		C				0201 000
C	RVS(I)=-RVK0(I)		0146 000		C	END IF			0202 000
C	RFO1=RFO1+RVK0(I)*RVS(I)		0147 000		C				0203 000
80	CONTINUE		0148 000		C	IF(RF GE RFO)THEN			0204 000
C			0149 000		C				0205 000
C	RLAMDA=2 0		0150 000		C	DO 120 I=1,N			0206 000
C			0151 000		C	RVS(I)=0 5*RVS(I)			0207 000
C	IF(4 0*RFO LT -RFO1)THEN		0152 000		120	CONTINUE			0208 000
C			0153 000		C				0209 000
C	DO 90 I=1,N		0154 000		C	RFO1=0 5*RFO1			0210 000
C	RVS(I)=-RVS(I)*4 0*RFO/RFO1		0155 000		C	RLAMDA=0 5			0211 000
90	CONTINUE		0156 000		C	GO TO 2			0212 000
C			0157 000		C	END IF			0213 000
C	RFO1=-4 0*RFO		0158 000		C				0214 000
C	END IF		0159 000		C	*****			0215 000
C			0160 000		C				0216 000
C	*****		0161 000		C	STEP NO 3			0217 000
C			0162 000		C				0218 000
C	STEP NO 2		0163 000		3	CONTINUE			0219 000
C			0164 000		C				0220 000
2	CONTINUE		0165 000		C	CALL GRAD1(RVX,N,RVG,ESP L SYS)			0221 000
C			0166 000		C				0222 000
C	UPDATE X-VALUES		0167 000		C	J-TRANPOSE			0223 000
C			0168 000		C				0224 000
C	DO 100 I=1,N		0169 000		C	DO 121 I=1,N			0225 000
C	RVX(I)=RVX0(I)		0170 000		C	DO 121 J=1,N			0226 000
C			0171 000		121	RMJT(I,J)=RMJ(J,I)			0227 000
C	DO 100 J=1,N		0172 000		C				0228 000
C			0173 000		C	K-VECTOR			0229 000
C	RVX(I)=RVX(I)+RMJ(I,J)*RVS(J)		0174 000		C				0230 000
100	CONTINUE		0175 000		C	DO 125 I=1 N			0231 000
C			0176 000		C	RVK(I)=0 0			0232 000
C			0177 000		C	DO 125 J=1,N			0233 000
C	IF(-RFO1 LE REPSILON)GO TO 999 ' EXIT *****		0178 000		C	RVK(I)=RVK(I)+RMJ(I J)*RVG(J)			0234 000
C			0179 000		125	CONTINUE			0235 000
C	IF(IX GE NITER)GO TO 999		0180 000		C				0236 000
C	IX=IX+1		0181 000		C	F			0237 000

04/11/84	16 59 16	TASK # 16000084	AMOS	GOULD S E L MPX-3	04/11/84	16 59 16	TASK # 16000084	AMOS	GOULD S E L MPX-3
C				0238 000			DO 170 I=1,N		0294 000
	RF1=0 0			0239 000			RTMP=RTMP+RVW(I)*RVW(I)/RFM		0295 000
	DO 130 I=1,N			0240 000	170		CONTINUE		0296 000
	RF1=RF1+RVK(I)*RVS(I)			0241 000	C				0297 000
130	CONTINUE			0242 000	C				0298 000
C				0243 000			DO 180 I=1 N		0299 000
C				0244 000			RVU(I)=RVW(I)-RVM(I)*RTMP		0300 000
C	B(0)			0245 000	180		CONTINUE		0301 000
C				0246 000	C				0302 000
C	RBO=RF1-RFO1			0247 000	C				0303 000
C				0248 000			RMTU=0 0		0304 000
C	! CALC M, X, KO			0249 000			RMUTU=0 0		0305 000
C				0250 000	C				0306 000
	DO 140 I=1,N			0251 000			DO 190 I=1 N		0307 000
	RVM(I)=RVS(I)+RVKO(I)-RVK(I)			0252 000			RMTU=RMTU+RVM(I)*RVU(I)		0308 000
	RVXO(I)=RVX(I)			0253 000			RMUTU=RMUTU+RVU(I)*RVU(I)		0309 000
	RVKO(I)=RVK(I)			0254 000	190		CONTINUE		0310 000
140	CONTINUE			0255 000	C				0311 000
C				0256 000			RMTU=1 0E06*RMTU**2		0312 000
C				0257 000			RMUTU=RFM*RMUTU		0313 000
	RFO=RF			0258 000	C				0314 000
	RFO1=RF1			0259 000	C				0315 000
C				0260 000			IF(RMTU LT RMUTU)THEN		0316 000
C	IF(RBO LT REPSILON)THEN			0261 000	C		! CAL VECTOR N STEP 4A		0317 000
				0262 000			UTS=0 0		0318 000
	DO 150 I=1,N			0263 000			UTU=0 0		0319 000
	RVS(I)=RVS(I)*RLAMDA			0264 000	C				0320 000
150	CONTINUE			0265 000			DO 200 I=1,N		0321 000
	RFO1=RFO1*RLAMDA			0266 000			UTS=UTS+RVU(I)*RVS(I)		0322 000
	GO TO 2			0267 000			UTU=UTU+RVU(I)*RVU(I)		0323 000
	END IF			0268 000	200		CONTINUE		0324 000
C				0269 000	C				0325 000
C	*****			0270 000			DO 210 I=1,N		0326 000
C				0271 000			RVN(I)=RVU(I)*UTS/UTU		0327 000
C	STEP NO 4			0272 000	210		CONTINUE		0328 000
C				0273 000	C				0329 000
4	CONTINUE			0274 000			RFN=UTS**2/UTU		0330 000
C				0275 000	C				0331 000
	RFM=0 0			0276 000			ELSE		0332 000
	DO 155 I=1,N			0277 000	C				0333 000
	RFM=RFM+RVM(I)*RVM(I)			0278 000			DO 220 I=1,N		0334 000
155	CONTINUE			0279 000			RVN(I)=0 0		0335 000
C				0280 000	220		CONTINUE		0336 000
C	IF(RFM LE REPSILON)GO TO 1			0281 000			RFN=0 0		0337 000
				0282 000	C				0338 000
	RFMU=0 0			0283 000			END IF		0339 000
	DO 160 I=1,N			0284 000	C				0340 000
	RFMU=RFMU+RVM(I)*RVS(I)			0285 000	C		*****		0341 000
160	CONTINUE			0286 000	C				0342 000
C				0287 000			STEP NO 5		0343 000
C	RFNU=RFMU-RFM			0288 000	C				0344 000
C				0289 000	C				0345 000
C				0290 000	5		CONTINUE		0346 000
C	CALC U			0291 000	C				0347 000
C				0292 000			RFB=RFN+RFNU*RFMU/RFM		0348 000
C	RTMP=0 0			0293 000	C				0349 000



04/11/84	16 59 16	TASK # 160000B4	AMDS	GOULD S E L MPX-3,	04/11/84	16 59 16	TASK # 160000B4	AMDS	GOULD S E L MPX-3,
		IF(RFB LT REPSILON)THEN		0350 000					0406 000
C		DO 230 I=1,N		0351 000	C		GTKO=0 0		0407 000
				0352 000			DO 250 I=1,N		0408 000
C				0353 000	250		GTKO=GTKO+RVG(I)*RVKO(I)		0409 000
230		RVN(I)=RVS(I)-RVM(I)*RFMU/RFM		0354 000	C				0410 000
		RFN=RBO-RFNU*RFMU/RFM		0355 000			DO 260 I=1 N		0411 000
		RFB=RBO		0356 000			RVKO(I)=RVKO(I)+RVP(I)*GTKO		0412 000
		END IF		0357 000	260		CONTINUE		0413 000
C				0358 000	C				0414 000
C		*****		0359 000	C				0415 000
C				0360 000	C		UPDATE J-MATRIX		0416 000
C		STEP NO 6		0361 000	C				0417 000
C				0362 000			DO 270 I=1 N		0418 000
6		CONTINUE		0363 000			DO 270 J=1 N		0419 000
C				0364 000			SMGK(I,J)=RVG(I)*RVP(J)		0420 000
C		IF(RFNU*RFMU LT RFM*RFN)THEN		0365 000	270		CONTINUE		0421 000
				0366 000	C				0422 000
		RFA=RFB-RFNU		0367 000	C				0423 000
		RFC=RFB+RFMU		0368 000			DO 300 I=1,N		0424 000
C				0369 000			DO 300 J=1,N		0425 000
		RFGAMA= SQRT( (1 0-RFMU*RFNU/(RFN*RFM))/(RFA*RFB))		0370 000			D=RMJ(I,J)		0426 000
C				0371 000			DO 290 K=1 N		0427 000
C		RFDELTA= SQRT(RFC/RFA)		0372 000	290		D=D+RMJ(I,K)*SMGK(K,J)		0428 000
				0373 000			RMJ(I,J)=D		0429 000
C		IF(RFC LT RFA)RFGAMA=-RFGAMA		0374 000	300		CONTINUE		0430 000
		ELSE		0375 000	C				0431 000
		RFGAMA=0 0		0376 000	C				0432 000
		RFDELTA= SQRT(RFMU/RFNU)		0377 000			IF(RFN LE 0 000)THEN		0433 000
		END IF		0378 000	C				0434 000
C				0379 000			DO 310 I=1,N		0435 000
C				0380 000			RVW(I)=RVKO(I)		0436 000
C		*****		0381 000	310		CONTINUE		0437 000
C				0382 000			END IF		0438 000
C		STEP NO 7		0383 000					0439 000
C				0384 000			GO TO 1 ' LOOP BACK-----		0440 000
		RFALFA=RFMU+RFNU*RFDELTA+RFM*RFN*RFGAMA		0385 000	C				0441 000
C				0386 000	999		CONTINUE ' ***** EXIT *****		0442 000
C		P - VECTOR		0387 000	C				0443 000
C				0388 000	C		WRITE(1,1100)IX,RF		0444 000
		DO 240 I=1,N		0389 000	C				0445 000
		RVP(I)=RVM(I)*(RFDELTA-RFN*RFGAMA)+RVN(I)*RFGAMA*RFMU		0390 000	C100		FORMAT(/ 5X, ROUTINE TERMINATED ITER #= I3 3X F = D18 10)		0446 000
C				0391 000	C		WRITE(1,1060)		0447 000
C		G-VECTOR		0392 000	C		WRITE(1,1012)(RVX(I) I=1 N)		0448 000
C				0393 000	C				0449 000
		RVG(I)=RVM(I)*(1 0+RFN*RFGAMA)/RFALFA		0394 000	C				0450 000
X		-RVN(I)*RFGAMA*RFNU/RFALFA		0395 000			RETURN		0451 000
				0396 000			ENTRY GNDV2		0451 010
C		H-VECTOR		0397 000	C				0451 020
C				0398 000	C				0451 030
		RVW(I)=RVM(I)*RFN*(1 0+RFGAMA*RFNU*RFMU/RFALFA)		0399 000	C		FIND MACHINE EPSILON		0451 040
X		-RVN(I)*(1 0+RFDELTA)*RFNU*RFMU/RFALFA		0400 000	C				0451 050
C				0401 000	C				0451 060
240		CONTINUE		0402 000			X=1 0		0451 070
C				0403 000	10		X=X*0 5		0451 080
C				0404 000			A=1 0+X		0451 090
C		UPDATE VECTOR K(0)		0405 000			B=A-1 0		0451 100

04/11/84 16 59 16 TASK # 16000084

AMOS

GOULD S E L MPX-32

```
IF(B NE 0 0)GO TO 10
EPS=X*2 0
ESP= SQRT(EPS)
C
  RETURN
END
*A1 LIB=HYDROLIB,,U
*A1 DIR=HYDRODIR,,U
*LIBED
*EOJ
**
0451 110
0451 120
0451 130
0451 140
0451 150
0452 000
0453 000
0454 000
0455 000
0456 000
0457 000
```

04/11/84	17 00 18	TASK # 16000084	AMOS	GOULD S E L MPX-3:	04/11/84	17 00 18	TASK # 16000084	AMOS	GOULD S E L MPX-3:
*JOB B FUNCT2 AMOS SLOF=L FUNCT2				0001 000					0081 000
*OPTION 2 3 4 5				0002 000	C	CALL RK41			0082 000
*FORT77				0003 000		RF=0 0			0083 000
C				0004 000		DO 10 I=1,N			0084 000
C				0005 000		DO 10 K=1,JA			0085 000
C				0006 000	C	RF=RF+RGF2(I)*(YX(I K)-RREF1(I,K))**2			0086 000
C	FUNCT1-MINIMIZES ERROR TO ESTABLISH SYSTEM PARAMETERS			0007 000		CONTINUE			0087 000
C	FUNCT2-MINIMIZES ERROR TO ESTABLISH GAIN VALUES			0008 000	10				0088 000
C				0009 000	C				0088 100
C	*****			0010 000	C				0088 200
C				0011 000	C				0088 300
C	SUBROUTINE FUNCT1(RF)			0012 000		RETURN			0089 000
C				0013 000		END			0090 000
C				0014 000		\$A1 LIB=HYDROLIB,.U			0091 000
C	INCLUDE C CONTR2			0015 000		\$A1 DIR=HYDRODIR,.U			0092 000
C				0016 000		\$LIBED			0093 000
C	REAL*4 RF ' RETURNED VALUE OF FUNCTION			0017 000		\$EDJ			0094 000
C				0019 000		**			0095 000
C				0019 100					
C				0019 200					
C				0019 300					
C	BEGX=RXR(JL-JL/5+1)			0020 000					
C	DO 5 I=1,N+1			0021 000					
C	Y(I)=RYXACT(I,1)			0022 000					
5	CONTINUE			0023 000					
C	LPLOT= FALSE			0024 000					
C	CALL RK41 ' ***** INTEGRATION ROUTINE*****			0025 000					
C	RF=0 0			0026 000					
C	DO 10 I=1,N			0027 000					
C	DO 10 K=1,(JA-1)			0028 000					
C	RF=RF+RGF1(I)*(YX(I,K)-RYXACT(I,K+1))**2			0029 000					
10	CONTINUE			0030 000					
C				0030 100					
C				0030 200					
C				0030 300					
C	RETURN			0031 000					
C	END			0032 000					
C	*****			0033 000					
C				0034 000					
C				0035 000					
C	SUBROUTINE FUNCT2(RF)			0063 000					
C				0064 000					
C	INCLUDE C CONTR2			0065 000					
C				0066 000					
C	LOCALS			0067 000					
C	REAL*4 RF ' RETURNED VALUE OF FUNCTION			0068 000					
CC				0070 000					
C				0071 000					
C	BEGX=RBEGX+DELTA			0072 000					
C	ENDX=BEGX+0 2*DELTA			0073 000					
C				0074 000					
C	DO 5 I=1,N+1			0075 000					
C	Y(I)=ENDY(I)			0076 000					
5	CONTINUE			0077 000					
C				0078 000					
C	LPLOT= FALSE			0080 000					

```

04/11/84 17 00 35 TASK # 16000084 AMOS GOULD S E L MPX-3
%JOB B DERFN2 AMOS SLOF=L DERFN2
$OPTION 2 3 4 5
$FORT77
C
C TWO SUBROUTINES USED TO CALCULATE THE NONLINEAR STATE
C EQUATIONS OF MOTION FOR ONE-DEGREE-OF-FREEDOM MOTION
C SIMULATOR SUBROUTINE DERFUN2-'PLANT' MODEL,
C SUBROUTINE DERFUN1-UPDATE MODEL
C
C *****
C
C SUBROUTINE DERFUN2(RCONT,N,V,DV,RFM) 'PLANT' MODEL
C
C DECLARE LOCALS
C
C INTEGER*2 N 'NUMBER OF STATES
C REAL*4 RFM(2) 'PARAMETERS
C REAL*4 RCONT 'CONTROL VARIABLE MA
C REAL*4 V(4) 'STATE VARIABLES
C REAL*4 DV(4) 'STATE EQUATION
C REAL*4 BETA 'BULK MODULUS OF FLUID
C REAL*4 XMAX 'MAX PISTON TRAVEL
C REAL*4 A1 'AREA #1, SQ INCH
C REAL*4 A2 'AREA #2 SQ INCH
C REAL*4 RMASS 'MASS, LB-SEC 2/INCH
C REAL*4 VOL1 ' #1 VOLUME, CU INCH
C REAL*4 VOL2 ' #2 VOLUME, CU INCH
C REAL*4 RG1 ' #1 PORT FLOW, CU IN/SEC
C REAL*4 RG2 ' #2 PORT FLOW, CU IN/SEC
C REAL*4 RPS 'SUPPLY PRESSURE, PSI
C
C BETA=1.5E05
C XMAX=30.0
C A1=8.0
C A2=8.0
C RMASS=8.634
C RPS=2000.0
C
C *****
C
C VOL1=A1*(XMAX+V(1)) 'VOLUME #1 OF ACTUATOR
C VOL2=A2*(XMAX-V(1)) 'VOLUME #2 OF ACTUATOR
C
C FLOW CALCULATIONS
C
0001 000
0002 000
0003 000
0004 000
0005 000
0006 000
0007 000
0008 000
0009 000
0010 000
0011 000
0012 000
0013 000
0014 000
0015 000
0016 000
0017 000
0018 000
0019 000
0020 000
0021 000
0022 000
0023 000
0024 000
0025 000
0026 000
0027 000
0028 000
0029 000
0030 000
0031 000
0032 000
0033 000
0034 000
0035 000
0036 000
0037 000
0038 000
0039 000
0040 000
0041 000
0042 000
0043 000
0044 000
0045 000
0046 000
0047 000
0048 000
0049 000
0050 000
0051 000
0052 000
0053 000
0054 000
0055 000
0056 000
RSIGN1=SIGN(1.0,RCONT) 'SIGN OF CONTROL VARIABLE
RSQ1=RPS*0.5*(0.5*RPS-V(3))*RSIGN1
RSGA1=ABS(RSQ1)
RSQ2=0.5*RPS-(0.5*RPS-V(4))*RSIGN1
RSGA2=ABS(RSQ2)
C
C FLOW TO PORT #1 OF ACTUATOR
C
C IF(RSGA1.NE.0.0)THEN
C RG1=0.172*RCONT*SQRT(RSGA1)*SIGN(1.0,RSQ1)
C ELSE
C RG1=0.0
C END IF
C
C FLOW TO PORT #2 OF ACTUATOR
C
C IF(RSGA2.NE.0.0)THEN
C RG2=0.172*RCONT*SQRT(RSGA2)*SIGN(1.0,RSQ2)
C ELSE
C RG2=0.0
C END IF
C
C STATE EQUATIONS *****
C
C DV(1)=V(2)
C DV(2)=(1.0/RMASS)*(V(3)*A1-V(4)*A2) 'LOAD EGN
C DV(3)=(BETA/VOL1)*(RG1-A1*V(2)) 'P(1)
C DV(4)=(BETA/VOL2)*(-RG2+A2*V(2)) 'P(2)
C
C RETURN
C END
C
C *****
C
C SUBROUTINE DERFUN1(RCONT,N,V,DV,RFM) 'UPDATED MODEL
C
C DECLARE LOCALS
C
C INTEGER*2 N 'NUMBER OF STATES
0057 000
0058 000
0059 000
0060 000
0061 000
0062 000
0063 000
0064 000
0065 000
0066 000
0067 000
0068 000
0069 000
0070 000
0071 000
0072 000
0073 000
0074 000
0075 000
0076 000
0077 000
0078 000
0079 000
0080 000
0081 000
0082 000
0083 000
0084 000
0085 000
0086 000
0087 000
0088 000
0089 000
0090 000
0091 000
0092 000
0093 000
0094 000
0095 000
0096 000
0097 000
0098 000
0099 000
0100 000
0101 000
0102 000
0103 000
0104 000
0105 000
0106 000
0107 000
0108 000
0109 000
0110 000
0111 000
0112 000

```

```

REAL*4 RFM(2)      STATE EQ PARAMETERS      0113 000
REAL*4 RCONT      CONTROL VARIABLE, MA     0114 000
REAL*4 V(4)       STATE VARIABLES          0115 000
REAL*4 DV(4)      STATE EQUATION           0116 000
REAL*4 XMAX       MAX PISTON TRAVEL        0117 000
REAL*4 A1         AREA #1, 6Q INCH         0118 000
REAL*4 A2         AREA #2, 5Q INCH         0119 000
REAL*4 RMASS      MASS, LB-SEC 2/INCH      0120 000
REAL*4 VOL1       #1 VOLUME, CU INCH       0121 000
REAL*4 VOL2       #2 VOLUME, CU INCH       0122 000
REAL*4 RQ1        #1 PORT FLOW, CU IN/SEC  0123 000
REAL*4 RQ2        #2 PORT FLOW, CU IN/SEC  0124 000
REAL*4 RPS        SUPPLY PRESSURE, PSI     0125 000
C
C
C
XMAX=30 0
A1=8 0
A2=8 0
RMASS=8 634
RPS=2000 0
C
C*****
C
C
VOL1=A1*(XMAX+V(1))
VOL2=A2*(XMAX-V(1))
C
C
FLOW CALCULATIONS
C
RSIGN1=SIGN(1 0,RCONT)  SIGN OF CONTROL VARIAB F 0144 000
RSQ1=RPS*0 5+(0 5*RPS-V(3))*RSIGN1 0145 000
RSGA1=ABS(RSQ1) 0146 000
RSQ2=0 5*RPS-(0 5*RPS-V(4))*RSIGN1 0147 000
RSGA2=ABS(RSQ2) 0148 000
C
C
FLOW TO PORT #1 OF ACTUATOR
C
C
C
IF(RSGA1 NE 0 0)THEN
RQ1=RFM(2)*RCONT*SQR1(RSGA1)*SIGN(1 0 RSQ1) 0156 000
ELSE
RQ1=0 0 0158 000
END IF 0159 000
C
C
FLOW TO PORT #2 OF ACTUATOR
C
C
C
IF(RSGA2 NE 0 0)THEN
RQ2=RFM(2)*RCONT*SQR1(RSGA2)*SIGN(1 0 RSQ2) 0166 000
ELSE
RQ2=0 0 0167 000

```

```

END IF 0169 000
C
C
C
STATE EQUATIONS *****
C
C
DV(1)=V(2) 0177 000
DV(2)=(1 0/RMASS)*(V(3)*A1-V(4)*A2)  LOAD EQN 0178 000
DV(3)=((1 0E06*RFM(1))/VOL1)*(RQ1-A1*V(2))  P(1) 0179 000
DV(4)=((1 0E06*RFM(1))/VOL2)*(-RQ2+A2*V(2))  P(2) 0180 000
C
C
RETURN 0181 000
END 0182 000
$A1 LIB=HYDROLIB,,U 0183 000
$A1 DIR=HYDRODIR U 0184 000
$LIBFD 0185 000
$EOD 0186 000
** 0187 000
0188 000
0189 000
0190 000

```

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04/11/84 17 01 06 TASK # 160000B4 AMOS GOULD S E L MPX-3 04/11/84 17 02 55 TASK # 160000B4 AMOS GOULD S E L MPX-3

$JOB B GRAD1 AMOS SLOF=L GRAD1
$OPTION 2 3 4 5
$FORT77
SUBROUTINE GRAD1(X,N,GRAD,ESP,LSYS)
REAL*4 X(20)
REAL*4 GRAD(20)
REAL*4 DELX
REAL*4 FK ' FUNCTION VALUE AT X(K)
REAL*4 FK1 ' FUNCTION VAL AT X(K+1)
REAL*4 ESP ' FOR DELTA X
INTEGER*2 N ' NO OF INDEPENDENT VARIABLES
LOGICAL*1 LSYS
C
C
C
CALL FUNCT (FK,X,N) ' FUNCTION VAL AT X(K)
IF(LSYS)THEN
CALL FUNCT1(FK) ' IDENTIFICATION
ELSE
CALL FUNCT2(FK) ' FEEDBACK
END IF
DD 20 IA=1,N
DELX=ESP
IF(X(IA) NE 0 0)DELX=DELX* ABS(X(IA))
X(IA)=X(IA)+DELX
C
CALL FUNCT (FK1,X,N) ' FUNCT AT X(K+1)
IF(LSYS)THEN
CALL FUNCT1(FK1) ' IDENTIFICATION
ELSE
CALL FUNCT2(FK1) ' FEEDBACK
END IF
GRAD(IA)=(FK1-FK)/DELX
X(IA)=X(IA)-DELX ' RESTORE X(K)
20 CONTINUE
RETURN
END
$A1 LIB=HYDROLIB, U
$A1 DIR=HYDRODIR U
$LIBFD
$EOJ
**
0000 100
0000 200
0000 300
0001 000
0002 000
0003 000
0004 000
0005 000
0006 000
0007 000
0008 000
0008 100
0009 000
0010 000
0011 000
0011 100
0011 200
0011 300
0011 400
0011 500
0012 000
0013 000
0014 000
0015 000
0016 000
0016 100
0016 200
0016 300
0016 400
0016 500
0017 000
0018 000
0019 000
0020 000
0021 000
0022 000
0023 000
0024 000
0025 000
0026 000
0026 000
0027 000
0028 000
0029 000
0030 000
0031 000
0032 000
0033 000
0034 000
0035 000
0036 000
0037 000
0038 000
0039 000
0040 000
0041 000

C
C
C
COMMON FOR MAIN
C
REAL*4 Y(5) ' STATES
REAL*4 DY(4) ' STATE EQUATIONS
REAL*4 RCONT ' CONTROL VARIABLE
REAL*4 RBEGX ' STORAGE FOR TIME
REAL*4 ENDX
REAL*4 DELX
REAL*4 DELTA ' SAMPLING INTERVAL
REAL*4 STRTY(5) ' START SEGMENT
REAL*4 ENDY(5) ' END SEGMENT
REAL*4 RK(4) ' STATE FEEDBACK COEFFICIENTS
REAL*4 RREF(5) ' REFERENCE SIGNAL INPUT
REAL*4 RREFX(5) ' STATIC REFERENCE FOR CONTROL
REAL*4 RFM(2) ' STATE PARAMETERS(VARIABLES)
REAL*4 RREFC(5,15) ' REFERENCE FOR CONTROL
REAL*4 RTANG(5) ' SLOPE FOT REFERENCE, CONTR
REAL*4 X ' TIME
REAL*4 RGF1(4) ' WEIGHTING FACTORS, IDENTIF
REAL*4 RGF2(4) ' WEIGHTING FACTORS, CONTROL
REAL*4 RREFI(5 60) ' GREED RFFERENCE ACCUMULATED
REAL*4 RYXACT(5 15) ' GREED POINT STATE VALUES
REAL*4 RXR(60) ' GREED POINTS TIME
REAL*4 YX(5 60) ' GREED POINTS STATE VALUFS
INTEGER*2 JL ' MAX GREED POINTS
INTEGER*2 JA ' GREED POINTS COUNTER
INTEGER*2 N ' NUMBER OF STATES
LOGICAL*1 LPLOTT ' PLOT FLAG
LOGICAL*1 LINEAR ' LINEAR VERSION FLAG
C
C
COMMON/CONTR/Y,DY,RCONT, RBEGX, ENDX, DELX, DELTA
X STRTY, ENDY, RK, RREF, RREFX, RFM, RREFC, RTANG, X, RGF1, RGF2
X RREFI, RYXACT, RXR, YX
X JL, JA, N, LPLOTT, LINEAR
C
C
C

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$JOB B LXYPLT AMOS SLOF=L LXYPLT
$OPTION 2 5
$FORT77
SUBROUTINE LXYPLT ( IFUN, MINT, MAX1, MIN1, MAX2, MIN2, MAX3, MIN3,
1 , MAX4, MIN4, VALN1, VALN2, VALN3, VALN4, PLTINC )
C
C----- GENERAL PURPOSE ON-LINE PLOT ROUTINE FOR LXY11 PRINTER -----
C
C
C AUTHOR - MICHAEL I PILDITCH
C DATE - 16-AUG-82
C VERSION - 04
C
C DESCRIPTION -
C THIS SUBROUTINE IS DESIGNED AS A SIMPLE GENERAL PURPOSE
C GRAPHICS PROGRAM PROVIDING 'ON-LINE' TIME HISTORIES FOR UP TO
C FOUR VARIABLES. OUTPUT IS FORMATTED FOR THE PRINTRONICS (LXY11)
C TYPE PRINTER/PLOTTER ONLY.
C THERE ARE TWO MODES OF OPERATION - FIRST INITIALISATION
C THIS DRAWS AND ANNOTATES THE Y OR HORIZONTAL AXES MAKING FULL
C USE OF THE PAPER WIDTH, AND SECOND DYNAMIC WHICH SCALES AND
C PLOTS THE Y VALUES AS THEY ARE INPUT.
C MARKER FREQUENCY AND PAPER SCROLLING BETWEEN CONSEQUITIVE
C INPUTS MAY BE SPECIFIED BY THE USER. INTERPOLATION BETWEEN POINTS
C IS LINEAR AND EVERY TENTH MARKER SPANS THE FULL PAGE WIDTH TO
C ASSIST IN READING OF THE PRINTED OUTPUT. Y VALUES WHICH EXCEED THE
C PAPER MARGINS ARE FORCED TO THE NEAREST PERMISSABLE VALUE AT THE
C PAGE EDGE AND THEIR MARKERS REMOVED. THE X OR TIME AXIS IS
C AUTOMATICALLY DRAWN AT Y=0.0 IF THIS IS BETWEEN THE SPECIFIED MAX
C AND MIN VALUES OR ELSE IS ALLIGNED WITH THE LIMIT CLOSEST TO Y=0.0
C IF LESS THAN FOUR GRAPHS ARE SELECTED ALL VALUES PLACED
C IN THE UNUSED PARAMETERS ARE IGNORED.
C THIS ROUTINE WILL ACCEPT ANY SCROLL INTERVAL GREATER OR
C EQUAL TO 0.05 INCHES, HOWEVER BECAUSE THERE ARE PHYSICALLY 72
C DOT LINES PER INCH IN THE X DIRECTION THOSE INTERVALS WHOSE
C RECIPROCALLS ARE FACTORS OF 72 WILL BE ACCURATE WHILE OTHERS
C MAY BE IN ERROR BY UP TO 4/ DUE TO ROUNDING TO THE NEAREST DOT
C LINE.
C IN THE EVENT THAT THE FIRST CALL TO THIS ROUTINE IS NOT
C AN INITIALISATION MODE (1,2,3,4) SUBSEQUENT CALLS WILL
C RESULT IN NO OUTPUT AND CONTROL WILL IMMEDIATELY RETURN TO THE
C CALLING PROGRAM.
C-----
C
C FORMAT OF THE CALL TO THIS SUBROUTINE -
C
C CALL LXYPLT (IFUN, MINT, MAX1, MIN1, MAX2, MIN2, MAX3, MIN3,
1 MAX4, MIN4, VALN1, VALN2, VALN3, VALN4, PLTINC)
C
C-----
C INPUT PARAMETERS
C
C NAME TYPE DESCRIPTION TYPICAL VALUE
C
C IFUN INTEGER CONTROLS MODE OF OPERATION
C 0= PLOT Y VALUES
C 1= SET AXIS FOR ONE GRAPH
    0001 000 C
    0002 000 C
    0003 000 C
    0004 000 C
    0005 000 C
    0006 000 C
    0007 000 C
    0008 000 C
    0009 000 C
    0010 000 C
    0011 000 C
    0012 000 C
    0013 000 C
    0014 000 C
    0015 000 C
    0016 000 C
    0017 000 C
    0018 000 C
    0019 000 C
    0020 000 C
    0021 000 C
    0022 000 C
    0023 000 C
    0024 000 C
    0025 000 C
    0026 000 C
    0027 000 C
    0028 000 C
    0029 000 C
    0030 000 C
    0031 000 C
    0032 000 C
    0033 000 C
    0034 000 C
    0035 000 C
    0036 000 C
    0037 000 C
    0038 000 C
    0039 000 C
    0040 000 C
    0041 000 C
    0042 000 C
    0043 000 C
    0044 000 C
    0045 000 C
    0046 000 C
    0047 000 C
    0048 000 C
    0049 000 C
    0050 000 C
    0051 000 C
    0052 000 C
    0053 000 C
    0054 000 C
    0055 000 C
    0056 000 C
    2= SET AXES FOR TWO GRAPHS
    0057 000
    3= SET AXES FOR THREE GRAPHS
    0058 000
    4= SET AXES FOR FOUR GRAPHS
    0059 000
    0060 000
    MINT INTEGER NUMBER OF CALLS BEFORE OUTPUT OF A MARKER 10
    0061 000
    0062 000
    MAX1 REAL*4 MOST POSITIVE AXIS VALUE GRAPH 1 120.0
    0063 000
    0064 000
    MIN1 REAL*4 MOST NEGATIVE AXIS VALUE GRAPH 1 -11.0, 2.1
    0065 000
    0066 000
    MAX2 REAL*4 MOST POSITIVE AXIS VALUE GRAPH 2
    0067 000
    (IGNORED IF ONLY ONE GRAPH SELECTED)
    0068 000
    0069 000
    MIN2 REAL*4 MOST NEGATIVE AXIS VALUE GRAPH 2
    0070 000
    (IGNORED IF ONLY ONE GRAPH SELECTED)
    0071 000
    0072 000
    MAX3/MIN3 REAL*4 AS GRAPH 2 BUT IGNORED IF LESS THAN
    0073 000
    THREE GRAPHS SELECTED
    0074 000
    0075 000
    MAX4/MIN4 REAL*4 AS GRAPH 3 BUT IGNORED IF LESS THAN
    0076 000
    FOUR GRAPHS SELECTED
    0077 000
    0078 000
    VALN1 VALN4
    0079 000
    REAL*4 INPUT Y VALUES (RELEVANT VALUE IGNORED
    0080 000
    IF GRAPH NOT SELECTED)
    0081 000
    0082 000
    PLTINC REAL*4 SCROLL DISTANCE BETWEEN CALLS (INCHES) 0.1
    0083 000
    0084 000
    -----
    0085 000
    NOTE - THIS SUBROUTINE MUST BE CATALOGUED WITH THE FOLLOWING
    0086 000
    ASSIGNMENT -
    0087 000
    A2 LP=SLO,20000
    0088 000
    0089 000
    0090 000
    -----
    0091 000
    0092 000
    0093 000
    LOGICAL*1 PLTBUF(132), INC(7)
    0094 000
    INTEGER*1 IBUF(132), IINC(7)
    0095 000
    EQUIVALENCE (IBUF(1), PLTBUF(1))
    0096 000
    EQUIVALENCE (IINC(1), INC(1))
    0097 000
    REAL FINC(6), MAX(4), MIN(4), VALD(4), VALN(4) AXIS(4)
    0098 000
    REAL MAX1, MAX2, MAX3, MAX4, MIN1, MIN2, MIN3, MIN4
    0099 000
    INTEGER FLAG, INDX(4), IFUN, TI, LP
    0100 000
    DATA IINC/ 1, 2, 4, 8, 16, 32, 63/
    0101 000
    DATA FINC/ 0.166666, 0.333333, 0.05, 0.066666, 0.083333, 0.1001/
    0102 000
    0103 000
    C SET LOGICAL UNIT NUMBERS
    0104 000
    0105 000
    DATA TI/ 1/
    0106 000
    DATA LP/ 3/
    0107 000
    0108 000
    VALN(1)=VALN1
    0109 000
    VALN(2)=VALN2
    0110 000
    VALN(3)=VALN3
    0111 000
    VALN(4)=VALN4
    0112 000

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04/11/84	17 01 27	TASK # 16000084	AMOS	GOULD S E L MPX-3	04/11/84	17 01 27	TASK # 16000084	AMOS	GOULD S E L MPX-3
C				0113 000					0169 000
		IF(IFUN EQ 0) GOTO 1000		0114 000			IF((IPOS+INDX(IGN)) LT 131) GOTO 1090		0170 000
		IF(IFUN EQ 1) GOTO 100		0115 000			PLTBUF(130)=PLTBUF(130) OR INC(6)		0171 000
		IF(IFUN EQ 2) GOTO 100		0116 000	1090		GOTO 1080		0172 000
		IF(IFUN EQ 3) GOTO 100		0117 000			CONTINUE		0173 000
		IF(IFUN EQ 4) GOTO 100		0118 000			IF((IPOS+INDX(IGN)) GT 1) GOTO 1100		0174 000
C				0119 000			PLTBUF(2)=PLTBUF(2) OR INC(1)		0175 000
		WRITE('UT',50)		0120 000	1100		GOTO 1080		0176 000
50		FORMAT(' LXYPLT- - ILLEGAL MODE')		0121 000	C		CONTINUE		0177 000
		RETURN		0122 000			PLTBUF(IPOS+INDX(IGN))=PLTBUF(IPOS+INDX(IGN)) OR INC(J)		0178 000
C				0123 000			IF (J EQ 7) IBUF(IPOS+INDX(IGN)) = IINC(J)		0179 000
C				0124 000	1080		CONTINUE		0180 000
C		DYNAMIC MODE		0125 000	1130		CONTINUE		0181 000
C				0126 000	C				0182 000
C				0127 000	1055		CONTINUE		0183 000
1000		CONTINUE		0128 000	C				0184 000
C				0129 000			IF(J EQ 7) MCNT=MCNT+1		0185 000
		IF(FLAG NE 1) RETURN		0130 000	C				0186 000
C				0131 000			IF(MCNT/10*10 NE MCNT OR J NE 7) GOTO 1110		0187 000
		MCOUNT=MCOUNT+1		0132 000			DD 1120 II=1,130		0188 000
C				0133 000			IBUF(II)= 63		0189 000
		IF(IRND NE 0) GOTO 1060		0134 000	1120		CONTINUE		0190 000
		NLPI=NLPI+1		0135 000	1110		CONTINUE		0191 000
1060		CONTINUE		0136 000	C				0192 000
C				0137 000			WRITE('LP',40) (PLTBUF(K),K=1,131)		0193 000
		DD 1050 L=1,NLPI		0138 000	40		FORMAT(1X,131A1)		0194 000
C				0139 000	C				0195 000
		DD 1010 I=1,130		0140 000	1050		CONTINUE		0196 000
		IBUF(I)= 64		0141 000	C				0197 000
1010		CONTINUE		0142 000			VALD(1)=VALN(1)		0198 000
		IBUF(131)= 5		0143 000			VALD(2)=VALN(2)		0199 000
C				0144 000			VALD(3)=VALN(3)		0200 000
		DD 1055 IGN=1,NGR		0145 000			VALD(4)=VALN(4)		0201 000
		DD 1130 JJ=1,2		0146 000	C				0202 000
C				0147 000			IRND=IRND-1		0203 000
		VAL=VALD(IGN)+(VALN(IGN)-VALD(IGN))/NLPI*L		0148 000	C				0204 000
C				0149 000			IF(IRND GE 0) GOTO 1070		0205 000
		IF(JJ EQ 2) VAL=AXIS(IGN)		0150 000			NLPI=NLPI-1		0206 000
C				0151 000			IRND=IRND		0207 000
		FPDS=(VAL-MIN(IGN))/(MAX(IGN)-MIN(IGN))*GW		0152 000	1070		CONTINUE		0208 000
C				0153 000	C				0209 000
		IF(FPDS GT 3200 0) FPDS = 3200 0		0154 000			RETURN		0210 000
		IF(FPDS LT -3200 0) FPDS =-3200 0		0155 000	C				0211 000
C				0156 000					0212 000
		IPOS=FPDS*10 0		0157 000					0213 000
		IF(FPOS LT 0 0) IPOS=IPOS-1		0158 000					0214 000
		DELPOS=(FPOS*10 0-IPOS)/10 0		0159 000	C				0215 000
C				0160 000	100		CONTINUE		0216 000
		J=1		0161 000	C				0217 000
1020		IF(FINC(J) GT DELPOS) GOTO 1030		0162 000			DD 140 I=1,130		0218 000
		J=J+1		0163 000			IBUF(I)= 64		0219 000
		GOTO 1020		0164 000	140		CONTINUE		0220 000
1030		CONTINUE		0165 000			IBUF(131)= 5		0221 000
C				0166 000	C				0222 000
		IF(MCOUNT/MINT*MINT EQ MCOUNT AND L EQ NLPI) J=7		0167 000			NGR=IFUN		0223 000
C				0168 000			MCOUNT=0		0224 000



	MCNT=0	0225 000	IF(NGR NE 3) GOTO 160	0281 000
C	VAL0(1)=VALN(1)	0226 000	GW=4 0	0282 000
	VAL0(2)=VALN(2)	0227 000	INDX(1)=3	0283 000
	VAL0(3)=VALN(3)	0228 000	INDX(2)=46	0284 000
	VAL0(4)=VALN(4)	0229 000	INDX(3)=89	0285 000
C		0230 000	DIV1=MIN(1)+(MAX(1)-MIN(1))/2 0	0286 000
	IF(PLTINC GT 0 0499) GOTO 300	0231 000	DIV2=MIN(2)+(MAX(2)-MIN(2))/2 0	0287 000
	WRITE('UT',310)	0232 000	DIV3=MIN(3)+(MAX(3)-MIN(3))/2 0	0288 000
310	FORMAT(' LXYPLT-- SCROLL INTERVAL TOO SMALL (MIN 0 05 INCHES) )	0233 000	WRITE('LP',110)MIN(1) DIV1,MAX(1) MIN(2),DIV2,MAX(2) MIN(3)	0289 000
	RETURN	0234 000	1 DIV3,MAX(3)	0290 000
300	CONTINUE	0235 000	110 FORMAT(1X,E12 4,4X,E12 4 4X E12 4 E12 4 3X,E12 4 4X F12 4	0291 000
	NLPI=72 0*PLTINC	0236 000	1 E12 4,4X,E12 4 3X,E12 4)	0292 000
	IROUND=1 0/(72 0*PLTINC-NLPI)	0237 000	GDTO 150	0293 000
	IRND=IROUND	0238 000	160 IF(NGR NE 2) GOTO 170	0294 000
C		0239 000	GW=6 0	0295 000
	MAX(1)=MAX1	0240 000	INDX(1)=5	0296 000
	MAX(2)=MAX2	0241 000	INDX(2)=70	0297 000
	MAX(3)=MAX3	0242 000	DIV=(MAX(1)-MIN(1))/3 0	0298 000
	MAX(4)=MAX4	0243 000	DIV1=MIN(1)+DIV	0299 000
	MIN(1)=MIN1	0244 000	DIV2=MIN(1)+DIV*2 0	0300 000
	MIN(2)=MIN2	0245 000	DIV=(MAX(2)-MIN(2))/3 0	0301 000
	MIN(3)=MIN3	0246 000	DIV3=MIN(2)+DIV	0302 000
	MIN(4)=MIN4	0247 000	DIV4=MIN(2)+DIV*2 0	0303 000
C		0248 000	WRITE('LP',111)MIN(1),DIV1,DIV2 MAX(1),MIN(2),DIV3,DIV4 MAX(2)	0304 000
	DO 320 I=1,NGR	0249 000	111 FORMAT(1X,E12 4,5X,E12 4,6X,E12 4,6X E12 4 E12 4 6X E12 4 6X	0305 000
	IF(MIN(I) LT MAX(I)) GOTO 320	0250 000	1 E12 4,6X,E12 4)	0306 000
	WRITE('UT',330) I	0251 000	GDTO 150	0307 000
330	FORMAT(' LXYPLT-- BAD LIMITS FOR GRAPH ,I2)	0252 000	170 CONTINUE	0308 000
	RETURN	0253 000	GW=12 0	0309 000
320	CONTINUE	0254 000	INDX(1)=6	0310 000
C		0255 000	DIV=(MAX(1)-MIN(1))/6 0	0311 000
	DO 270 I=1,NGR	0256 000	DIV1=MIN(1)+DIV	0312 000
	IF(MAX(I) GT 0 0) GOTO 280	0257 000	DIV2=MIN(1)+DIV*2 0	0313 000
	AXIS(I)=MAX(I)	0258 000	DIV3=MIN(1)+DIV*3 0	0314 000
	GDTO 270	0259 000	DIV4=MIN(1)+DIV*4 0	0315 000
280	CONTINUE	0260 000	DIV5=MIN(1)+DIV*5 0	0316 000
	IF(MIN(I) LT 0 0) GOTO 290	0261 000	WRITE('LP',112)MIN(1),DIV1,DIV2,DIV3,DIV4,DIV5,MAX(1)	0317 000
	AXIS(I)=MIN(I)	0262 000	112 FORMAT(1X,6(E12 4,7X),E12 4)	0318 000
	GDTO 270	0263 000	150 CONTINUE	0319 000
290	CONTINUE	0264 000	C	0320 000
	AXIS(I)=0 0	0265 000	DO 210 II=1,NGR	0321 000
270	CONTINUE	0266 000	PLTBUF(INDX(II))=PLTBUF(INDX(II)) OR INC(1)	0322 000
C		0267 000	IF(NGR NE 1) GOTO 220	0323 000
	IF(NGR NE 4) GOTO 180	0268 000	DO 230 III=1,6	0324 000
	GW=3 0	0269 000	PLTBUF(INDX(II)+III*20)-PLTBUF(INDX(II)+III*20) OR INC(1)	0325 000
	INDX(1)=3	0270 000	230 CONTINUE	0326 000
	INDX(2)=35	0271 000	220 CONTINUE	0327 000
	INDX(3)=68	0272 000	IF(NGR NE 2) GOTO 240	0328 000
	INDX(4)=100	0273 000	DO 250 III=1,3	0329 000
	WRITE('LP',113) MIN(1),MAX(1),MIN(2),MAX(2),MIN(3),MAX(3),	0274 000	PLTBUF(INDX(II)+III*20)=PLTBUF(INDX(II)+III*20) OR INC(1)	0330 000
1	MIN(4),MAX(4)	0275 000	250 CONTINUE	0331 000
113	FORMAT(1X,E12 4,8X,E12 4,1X,E12 4 8X,E12 4,1X,E12 4,6X	0276 000	240 CONTINUE	0332 000
	,E12 4,1X,E12 4,8X,E12 4)	0277 000	IF(NGR NE 3) GOTO 211	0333 000
	GDTO 150	0278 000	DO 260 III=1,2	0334 000
180	CONTINUE	0279 000	PLTBUF(INDX(II)+III*20)=PLTBUF(INDX(II)+III*20) OR INC(1)	0335 000
		0280 000	260 CONTINUE	0336 000

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04/11/84 17 01 27 TASK # 16000084 AMOS GOULD S E L MPX-3:
211 CONTINUE 0337 000
IF(NGR NE 4) GOTO 210 0338 000
PLTBUF(INDX(II)+GW*10 0)=PLTBUF(INDX(II)+GW*10 0) OR INC(1) 0339 000
210 CONTINUE 0340 000
C 0341 000
DO 120 I=1,7 0342 000
WRITE('LP',40)(PLTBUF(K),K=1,131) 0343 000
120 CONTINUE 0344 000
C 0345 000
DO 200 II=1,NGR 0346 000
DO 130 I=INDX(II),INDX(II)+GW*10 0-1 0347 000
IBUF(I)= 63 0348 000
130 CONTINUE 0349 000
200 CONTINUE 0350 000
C 0351 000
WRITE('LP',40)(PLTBUF(K),K=1,131) 0352 000
C 0353 000
FLAG=1 0354 000
C 0355 000
RETURN 0356 000
C 0357 000
END 0358 000
$A1 LIB=HYDROLIB,,U 0358 100
$A1 DIR=HYDRODIR, U 0358 200
$LIBED 0359 000
$EQJ 0360 000
** 0361 000

```

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04/11/84 17 03 08 TASK # 16000084 AMOS GOULD S E L MPX-3:
C 0001 000
C 0002 000
C COMMON AREA TO BE USED BY STATE EGN(B DERFUN) 0003 000
C 0004 000
REAL*4 FM(4,4) ' F-MATRIX 0005 000
REAL*4 GM(4) ' G-MATRIX, HERE ONE CONTROL 0006 000
COMMON/DERF/FM,GM 0007 000
C 0008 000
C STORED IN C DERF2 0009 000
C 0010 000
C 0011 000

```

```

04/11/84 17 02 16 TASK # 16000084 AMOS GOULD S E L MPX-3:
EXAMPLE FOR IMPLEMENTATION OF ADAPTIVE LINEAR CONTROL FEEDBACK 0000 100
TO CONTROL HIGHLY NON-LINEAR ELECTRO-HYDRAULIC SYSTEM 0000 200
AMOS BURSHTEIN BURTEK INC, DECEMBER 1983 0000 300
4 0000 400
0 0 0 0 0 0001 000
1 0 0 -5000 0 5000 00 0002 000
0 0 926 0 -21 5625 0 0 0003 000
0 0 -926 0 0 -21 5625 0004 000
0 0 0 0 1075 0 1075 0 0005 000
0 0 0 050 0 001 0 2 0006 000
0 0 0 0 1000 0 1000 0 0007 000
0 10 0 1 1295 -1295 0008 000
0 0 0 0 1000 0 1000 0 0009 000
1 0 1 0E-2 1 0E-4 1 0E-4 0010 000
0 0E-02 0 0E-02 1 0E00 1 0E00 0011 000
140 180 0012 000

```

```

04/11/84 17 03 25 TASK # 16000084 AMOS GOULD S E L MPX-3:
C 0001 000
C 0002 000
C C DERF1 0003 000
C 0004 000
C REAL*4 RFM(1) ' BULK MODULUS OF FLUID(1 5E5) 0005 000
REAL*4 RFM(2) ' VALVE GAIN ( 172) 0006 000
COMMON/DERF1/RFM 0007 000
C 0008 000
C 0009 000
C ***** 0010 000

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2  
VITA

Amos Burshtein

Candidate for the Degree of  
Doctor of Philosophy

**Thesis:** A SYNTHESIS METHOD FOR ADAPTIVE CONTROL OF  
NONLINEAR SYSTEMS WITH APPLICATION TO A ONE  
DEGREE-OF-FREEDOM MOTION SIMULATOR

**Major Field:** Mechanical Engineering

**Biographical:**

**Personal Data:** Born in Tel-Aviv, Israel, June 21,  
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**Education:** Received the Bachelor of Science degree in  
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**Professional Experience:** Supervisor, assembly line of  
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Engineer, material handling and recovery tank,  
Israeli Defense Forces, 1965-1968; Laboratory  
Engineer, American Hoist & Derrick Co., St. Paul,  
Minnesota, 1969-1970; Manager, Special Ground  
Support Equipment Engineering Department, Israel  
Aircraft Industries, 1972-1976; Manager, Logistic  
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Industries, 1977-1980; System Engineer, Burtek,  
Inc., Tulsa, Oklahoma, 1981 to present.